# BOUNDARY LAYER SIMILATOR IMPROVEMENT 

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## FOREWORD

1
This final report presents work conducted for the Marshall Space Filght Center (MSFC) in response to the requirements of Contract NAS8-35976. The work presented here was performed by REMTECH, Inc., Hunstville, Alabama and is titled, "Boundary Layer Simulator Improvement".

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LISTING OF THE UPDATED SUBROUTINES IN BLIMPJ

The primary goal of the work reported here was to improve the existing Boundary Layer Integral Matrix Procedure, Version J (BLIMPJ) ${ }^{1}$. BLIMPJ has been used in the industry as a rigorous boundary layer program in connection with the existing JANNAF reference programs such as $O D E$ and TDK ${ }^{2}$. It is capable of treating two-dimensional and axisymmetric nozzles with a variety of wall boundary conditions which include regenerative and transpiration cooling as well as ablating wall materlals. The improvements described herein have potential use In the design of the future Orbit Transfer Vehicle (OTV) engines.

The projected engine design for the OTV would utilize an expander cycle operation mode. In this mode, heat energy obtained through a regeneratively cooled wall is used to drive the turbines and pumps. $0_{2}-\mathrm{H}_{2}$ propellant system is used to react in the combustion chamber at pressure levels of 1500-2000 psia at a mixture ratio of 6. The reaction products are expanded through a nozzle of large area ratio, ranging from 400 to 3000. Although the above chamber pressures and $0 / F$ ratio for a $\mathrm{O}_{2}-\mathrm{H}_{2}$ system are not uncommon for the currently operating Space Shuttle maln engines (SSMEs), the area ratio is only of the order of 80. These high chamber pressure expander cycle engines depend primarily on the heat energy transmitted from the combustion products through the thrust chamber wall. The larger the regenerative heat transfer the higher the chamber pressure which in turn permits larger area ratio motors. These engines

1. Evans, R., "Boundary Layer Integral Matrix Procedure, BLIMP-J User's Manual," Aerotherm Division/Acurex Corporation, July 1975, under Contract NAS8-30930.
2. Nickerson, G.R., Coats, D.E., and Bartz, J.L., "The Two-Dimensional Kinetic (TDK) Reference Computer Program," Engineering and Programming Manual, Ultrasystems, Inc., December 1973, under Contract NAS9-12652.
and the associated interlor nozzle flowflelds are outside the range of current engineering experience. The heat transfer to the nozzle wall is affected by such varlables as wall roughness, relaminarization, and the presence of particles in the flow. The motor performance loss for these nozzles with thick boundary layers is inaccurate using the existing procedure coded in BLIMPJ. Flow expansion within large area ratio nozzles and assoclated low pressures and temperatures may produce two-phase flow conditions (liquid droplets or lce crystals) adjacent to the wall especially in connection with strongly cooled walls. The presence of such particles will have some effect on the friction and heat transfer mechanism within the boundary layer. Moreover, there are discussions in the technical community of replacing the nozzle wall around the throat by an ablative wall. This would reduce high heat-transfer to the nozzle throat because of ablation while introducing the ablation products in the nozzle boundary layer and the inviscid part of the nozzle flowfleld. All these modificatlons and innovations require investigations and Implementation in BLIMPJ code of the follow ing simplified analytical formulations:

- Wall surface roughness simulation and its impact on heat transfer and shear effects.
- Prediction of relaminarization regions with approximations on heat transfer and friction along the wall.
- Presence of particles in the boundary layer and their impact on heat transfer and friction.
- Re-evaluation of the existing boundary layer thrust loss calculation method for nozzles with large area ratios, experiencing thlck boundary layers at low density and high Mach number flow situations.

Various versions of BLIMPJ were recelved from Marshall Space FIIght Center (MSFC). Apart from the version avallable at REMTECH, a total of three additional versions including the (i) Aerotherm, (il) MSFC, and (III) mini-versions was
tABLE 1.1 bLIMPJ SUBROUTINES

obtalned. It was recommended by MSFC to use the mini-version for making modifications to the code. The mini-version is a cleaner and shorter version of the code and has fewer subroutines when compared with the Aerotherm version in Table 1.1. In order to access the code at various subroutines for moditications, a macro flow diagram was prepared and is provided in Fig. 1.1.

The varlous tasks described earlier are discussed in the following sections. Section 2 discusses the effects of wall roughness on skin friction and heat transfer. Section 3 highlights the mechanism and effects of relaminarization, whereas Section 4 discusses the effects of particles on skin friction and heat transfer rate on the nozzle wall. Section 5, on the other hand, focuses on the re-evaluation of the existing boundary layer thrust loss calculation method for nozzles with large area ratios experiencing thick boundary layers. The last four sections described above are self-contained in that the technical discussion for each item along with the corresponding figures and list of references are contalned in that section, independent of any other section. These sections also describe applications of the various modules in a composite fashion if more than one effect needs to be considered. Finally, Section 6 makes recommendations both in the areas of analytical and experimental techniques for future work.


Fig. 1.1 BLIMPJ Mini-Version Macro Flow Diagram

Section 2.0

## WALL SURFACE ROUGHNESS EFFECTS

### 2.1 Background

The importance of wall surface roughness which increases the resistance to fluld flows has been recognized for many years. One of the principal parameters influencing the surface heat transfer to a rough wall is the roughness helght, $k$.

The problem of modelling turbulent flow over rough surfaces has been divided into three regimes:

Regime 1: Smooth - The roughness size is so small that the protrusions are contained within the laminar sublayer. The surface skin friction and heat transfer are not changed from smooth surface values.

Regime 11: Iransitional - Some of the roughness elements protrude outside of the laminar sublayer. The skin friction and heat transfer are increased above the smooth surface values.

Regime 111: Eully Rough - All surface roughness elements protrude outside of the laminar sublayer. The increase in skin friction is primarily a result of form drag of the roughness elements.
H. Schlichting (Ref. 1) summarizes all the early work on rough wall measurements in turbulent flow and describes the evaluation of the "equivalent sand grain roughness height", $K_{s}$, which is based on the early work of Nikuradse (Ref. 2). Many theorles and correlations, following Nikuradse, employ the parameter $K_{s}$. Defining $K_{s}$ for a given surface condition is not a straightforward task. Schlichting (Ref. 1) describes procedures for a given array of roughness elements. Recently, Dirling (Ref. 3) has devised a correlation for $K_{s}$ and has applied it to the prediction of nosetip shape change. In modeling the effects of roughness on skin friction, the velocity proflle through the boundary layer has
been correlated with surface roughness of sand. Data and empirical correlations have been developed for other types of roughness elements to obtain the equivalent sand roughness. That is, the sand roughness which yields the same velocity profile is the roughness of interest. There is considerable uncertalnty in the determination of the equivalent sand roughness for roughness elements which are randomly shaped and spaced. Physical spacing, relating to the type of cavity flow that is established, the inclination of the roughness element surface to the flow direction, and the increased surface area are some of the important elements in the calculation of $K_{s}$. Figure 2.1 shows the correlation developed by Dirling (Ref. 3). The roughness density parameter $\Lambda$ is defined as shown on the figure, where $A_{s}$ is the windward surface area of the roughness, $A_{p}$ is the projected area of the roughness in the flow direction and $D$ is the inverse square root of the roughness elements per unit area. The correlation shown is derived from velocity measurements and is applicable for rough wall skin friction calculations. For the analysis given here, the $K_{s}$ parameter is not investigated, but instead, it is assumed that $\mathrm{K}_{\mathrm{s}}$ is given.

### 2.2 Roughness Options

The purpose of the task in this section is to determine which simplified correlations are appropriate for application in the BLIMPJ computer code. The correlations avallable in the Iiterature, which perform "point" calculations based on local edge and wall quantities, were reviewed. The slgnificance of "point" calculations lies in the fact that the history effects in the boundary layer at other points do not affect the calculation at the polnt under consideration. An excellent paper by Seidman (Ref. 4) reviewed some of these correlations and compared them with incompressible and compressible data. The approprlate options performing "point" calculations are given below:

| 2-0 ROO ELEMENTS | 2-O WAVY SURFACE |
| :---: | :---: |
| BETTERMANN LIUETAL Streeter | - traEETER HAUGHTON |
| 1-OELEMENTS - SCHLICHTING | 0 COHEN |


| Skin Friction Options | Heat Transfer Options |
| :--- | :---: |
| 1. Prandtl-Schlichting | 1. Seidman |
| 2. Droblenkov | 2. Hill |

The mathematical expressions are given in Ref. 4. There are two options for calculating skin friction and four possible combinations that can be used to calculate heat transfer rate. For reasons described in the next subsection, Hill's correlation was not coded in BLIMPJ. As a result, only two combinations for heat-transfer rate calculation remalned. The mathematical expressions for the above options were taken from Ref. 4 and are listed in Table 2.1 along with the input-output variable list that is used in the roughness subroutine. The expression (A.1) in Table 2.1 contain the calculation of a compressibility facfor in terms of the enthalpy ratio. Although, in the original paper (Ref. 4) the corresponding temperature ratios are chosen, it is customary to use the enthal py ratios instead of temperature ratio in order to include real gas effects. This would be approprlate for the $\mathrm{O}_{2}-\mathrm{H}_{2}$ reactive system to be used in the future OTV motor, where the combustion temperatures are in the order of $6000^{\circ} \mathrm{R}$ and real gas effects exist.

Another option by Cebeci was selected to slmulate the effects of a rough wall on the boundary layer and to account for "history" effects in the boundary layer. In Ref. 8, the turbulent mixing length of the eddy viscosity expression Is modifled for the inner region of a two-layer turbulence model to include the effects of surface roughness. Assuming that the velocity profiles for smooth and rough walls are similar, the expression for the mixing length given by

$$
\begin{equation*}
l=0.4 y\{1-\exp (-y / A)\} \tag{2.1}
\end{equation*}
$$

is modifled and rewritten as,

TABLE 2.1
ROUGH WALL HEAT TRANSFER OPTIONS

## Options 1 and 2:

Skin friction compressibility (Young)

$$
\begin{equation*}
\frac{C_{f}}{C_{f i}}=0.365\left(\frac{H_{e}}{H_{a w}}\right)+0.635\left(\frac{H_{e}}{H_{w}}\right) \tag{A.1}
\end{equation*}
$$

Incompressible rough wall skin friction
Option (1) Prandtl-Schlichting

$$
\begin{equation*}
c_{f i}=\left[2.87+1.58 \log _{10}(x / k)\right]^{-2.5} \tag{A.2}
\end{equation*}
$$

Option (2) Droblenkov

$$
\begin{equation*}
c_{f i}=0.0139(x / k)^{-1 / 7} \tag{A.3}
\end{equation*}
$$

Rough surface turbulent Stanton Number (Seidman)

$$
\begin{equation*}
S t=\frac{C_{f}}{2}\left[1+A\left(\frac{C_{f}}{2}\right)^{0.725}\left(R_{k}\right)^{0.45}(\operatorname{Pr})^{0.8}\right]^{-1} \tag{A.3}
\end{equation*}
$$

where $A=0.52$ nominal and range from 0.45 to 0.7 (Owen \& Thomson), and $C_{f}$ is obtained from Equ. (A.1).

Transition criterion (Fenter)

$$
\begin{equation*}
n_{k}=\frac{\rho_{W} U_{\tau} k}{u_{w}} \text { where } U_{\tau}=U_{e} \sqrt{\frac{C_{f}}{2} \frac{\rho_{e}}{\rho_{w}}} \tag{A.4}
\end{equation*}
$$

$$
\begin{aligned}
\quad n_{k} \leqslant 5 & \text { Smooth } \\
5 \leqslant n_{k} \leqslant 100 & \text { Transitionally rough } \\
100 \leqslant n_{k} & \text { Rough }
\end{aligned}
$$

TABLE 2.1 (Continued)

## INPUT VARIABLES

$$
\begin{aligned}
& X=\text { Running length (ft) } \\
& k=\text { Sand roughness height (ft) } \\
& H_{a w}=\text { Aidabatic wall enthalpy (Btu/lbm) } \\
& H_{e}=\text { B.L. edge enthalpy (Btu/lbm) } \\
& H_{W}=\text { Wall enthalpy (Btu/lbm) } \\
& \rho_{e}=\text { B.L. edge density ( } 1 \mathrm{bm} / \mathrm{ft}^{3} \text { ) } \\
& \rho_{\mathrm{W}}=\text { Wall density ( } 1 \mathrm{bm} / \mathrm{ft}^{3} \text { ) } \\
& \mu_{W}=\text { Wall viscosity (1bm/ft-sec) } \\
& \mu_{e}=\text { Edge viscosity ( } 1 \mathrm{bm} / \mathrm{ft}-\mathrm{sec} \text { ) } \\
& \operatorname{Pr}=\operatorname{Prandt} 1 \text { number (Edge) } \\
& \text { ICF }=\text { Skin friction flag } 1 \ldots \text { Prandtl-Schlichting } \\
& S t_{s}=\text { Smooth wall Stanton number } \\
& U_{e}=\text { B.L. edge velocity ( } \mathrm{ft} / \mathrm{sec} \text { ) }
\end{aligned}
$$

$$
\begin{aligned}
C_{f} & =\text { Rough wall sking friction coefficient } \\
S t & =\text { Rough wall Stanton number } \\
P C T & =\text { Percent of transition to fully rough }
\end{aligned}
$$

$$
\begin{equation*}
l=0.4(y+\Delta y)[1-\exp \{-(y+\Delta y) / A\}] \tag{2.2}
\end{equation*}
$$

where the coordinates are displaced by an amount $\Delta y$. He expresses $\Delta y$ as a function of an equivalent sand-grain roughness parameter $K_{s}^{+}\left(\equiv K_{s} U_{\tau} / \nu\right), 1, e_{1}$,

$$
\begin{equation*}
y=0.9\left(V / U_{\tau}\right)\left\{\sqrt{K_{s}^{+}}-K_{s}^{+} \exp \left(-K_{s}^{+} / 6\right)\right\} \tag{2.3}
\end{equation*}
$$

This expression is valid for $4.535<K_{s}^{+}<2000$, with the lower limit corresponding to the upper bound for a hydraulically smooth surface.

### 2.3 Examples

In order to lllustrate the valldity of the roughness options against measured data, first the skin-friction and heat-transfer data were collected from the original report by Pimenta, Moffat and Kays (Ref. 5). The two sets of data collected were for flat plates at moderate freestream velocities. Since BLIMPJ could not be run for external flow situations, BLIMPK (appllcable for external flow) was modifled to include the roughness options 1 and 2 and was run for an equivalent sand roughness of $K_{s}=.002583 \mathrm{ft}$. employing the only-resident, Kendall's turbulence model. Figure 2.2 contains the two cases for which the two skin-friction options were used. It is seen that the two options bracket the data, although Droblenkov's approach is closer to the data. Figure 2.3, on the other hand, shows the heat-transfer computations based on Seidman's Stanton number correlation. Again, the two skin-friction optlons along with Seidman's heat transfer correlation bracket the heat-transfer data, although one combination seems to predict the data better than the other one. Another Stanton number correlation by Hill was checked out (Fig. 2.4a), by varying the value of A in Hill's correlation. It is found that Hill's correlation underpredicts the data considerably. Figure 2.4b, on the other hand, gives comparison of



Fig. 2.3 Comparison Of Stanton Number Correlation With Data


(B)

Fig. 2.4 Comparison Between Stanton Number Correlation And Data With Varlable Parameter A

Seidman's correlation with heat-transfer data for three values of $A$. The nominal value of 0.52 for $A$ seems to predict the data quite well.

The rationale for checking the roughness heat-transfer options against data In external flow is a result of little or no data being avallable for nozzles having rough walls. Some roughness data obtained in an MSFC test on a 40-K subscale regenatively cooled nozzle (Ref. 6) were communicated to the authors. On closer examination, it was found, however, that the nozzle was rough at the throat region only. In other words, the equivalent sand roughness is not constant throughout the nozzle and none of the roughness options described here applies to such a situation. Moreover, the concept of equivalent sand roughness breaks down, since similarity in the boundary layer can no longer be satisfied. Instead, to exercise the three roughness options in BLIMPJ, the code was modifled to integrate all the options. In the meantime, the geometry package of a generic OTV nozzle was received (Ref. 7) along with the wall temperatures and wall pressures (given In Fig. 2.5). The code was first checked out for the OTV smooth wall situation using two different turbulence models including the Kendall and Cebeci-Smith models. The heat transfer distributions on the nozzle wall are given in Fig. 2.6. As noted by other Investigators, the Cebeci-Smith model predicted lower heating rates. A fictitious value of the equivalent sand roughness of 0.00125 ft . was used to run BLIMPJ for the OTV nozzle using first the roughness option 3 (which used a modification to Cebeci-Smith turbulence model). An example of the namelist tape for BLIMPJ using a roughness option is given in Table 2.2. The heat-transfer results are plotted in Fig. 2.9 and compared with those for a smooth wall. Heat rates are approximately 3 times higher for the rough wall than for the smooth wall in the peak heating region occurring around the throat. Although the skin friction and heating rate values are quite high for a rough nozzle locally in the throat region, the integrated values of


Fig. 2.5 Input Wall Pressure And Temperature Variation For A Typical OTV Nozzle

```
FEMTEOFHINC.
```



Fig. 2.6 Comparison Of Two Turbulence Models

TABLE 2.2 Example Of Namelist Input For Roughness Option

these quantities over the whole nozzle in relation to the smooth wall values are much lesser in magnitude. Since this roughness option modifles the turbulence model due to the presence of roughness, Fig. 2.7a was prepared to compare the velocity profiles between the rough and smooth wall cases at the nozzle throat. Figure 2.7 b , on the other hand, compares the velocity proflles given in normalized $y$-coordinates. It is clearly seen from both the plots that not only the boundary layer is thicker but is pushed upward as suggested by Cebeci. This phenomenon has also been observed experimentally be Voisinet (Ref. 9) and is reproduced in Fig. 2.7c as evidence.

The other two roughness options were also exerclsed for the same OTV nozzle with the above equivalent sand roughness helght. Since the enthalples in the expression (A1) in Table 2.1 are with respect to $T=0^{\circ} \mathrm{R}$ as the reference, the concept was modified in BLIMPJ to integrate $C_{p}$ with respect to $T$ from $T=0^{\circ} R$ to either the wall or the edge temperature to calculate $H_{w}$ or $H_{e}$, respectively. Noting that $C_{p}$ is calculated as a function of $T$ in the boundary layer, an extrapolation was made on $C_{p}$ to a value down to $T=0^{\circ} R$ as shown in Fig. 2.8 for the OTV nozzle throat location. A numerical intergration was performed within the code to calculate all the required enthalpies, and consequently, to compute skin friction and heat transfer rates. Figures 2.9, 2.10 and 2.11 compare heat flux, Stanton number and skin friction coefficient distribution using all the three avallable roughness options with $K_{s}=0.00125 \mathrm{ft}$. The comparison among the three optlons is quite reasonable near the throat and downstream of the throat. However, some disparities remaln in Stanton number and skin friction in the subsonic contraction section of the nozzle, particulary for Options 1 and 2.


Fig. 2.7a Comparison Of Velocity Distribution Between Rough And Smooth Walls At The OTV Nozzle Throat


Fig. 2.7b Comparison Of Velocity Distribution Between Rough And Smooth Walls At The OTV Nozzle Throat


$$
\begin{aligned}
& K=\text { Equivalent Sand Roughness Height } \\
& \dot{m}=\text { Mass Transfer Rate }
\end{aligned}
$$

Fig. 2.7c Typical Velocity Profiles Given By Voisinet (Ref. 9)


Fig. 2.8 Variation Of Specific Heat of $\mathrm{H}_{2} / \mathrm{O}_{2}$ Reaction Products With Static Temperature At OTV Nozzle Throat


Fig. 2.9 Comparison Of Heat Flux Distribution On The Wall Of The OTV Nozzle Wall Using Various Roughness Options


Fig. 2.10 Comparison Of Stanton Number Distribution On The OTV Nozzle Wall Using Various Roughness Options


Fig. 2.11 Comparison Of Skin-Friction Coefficient Distribution On The OTV Nozzle Wall

### 2.4 Discussions

The correlations and modifications incorporated in BLIMPJ to account for roughness would be very good candidates for evaluating the thermal losses on the OTV nozzles. The results given in Figs. 2.9-2.11 for a fictitious sand roughness show that although the comparison of $\dot{q}$, St and $C_{f_{i}}$ on the OTV wall between the three options is reasonable, there is still about 20 to 30 percent variation In the peak heating areas of the nozzle. It must be noted that certaln engineering approximations have been incorporated in the evaluation of $H$ in the calculation of the compressiblity factor, $C_{f} / C_{f_{i}}$ in Options 1 and 2. The Option 3 , on the other hand, is a more systematic modification of the turbulence model to account for wall surface roughness. It not only gives the heat transfer at the wall, but also provides the detalls of the turbulence scale change effects within the boundary layer. The effects of wall roughness on the law-of-the-wall results have been noted by others (Ref. 8) to cause a downward shift in the profiles with increased roughness. This meant that for the same value of the law-of-the-wall coordinate, $y^{+}$, the velocity is lower. The same phenomenon was observed in the work presented earller. One item In the Cebeci roughness model (Ref. 8) is the upper limit of 2000 for the equivalent sand-graln roughness parameter, $\mathrm{K}_{\mathrm{s}}$ for which the modification of the length scale is valid. In the code modification, a value of 4000 was used for running the case presented earlier. The validity of this limit must be examined experimentally. Suggestions for future work in this area appear in Sec. 6.

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Section 3.0
RELAMINARIZATION

### 3.1 Background

The prediction of relaminarization phenomena is one of the strongest tests of validity of the turbulence models. Relaminarization is basically a reversion from turbulent to laminar boundary layer. Relaminarization is principally caused by severe flow acceleration effects that typically occur internally in the convergent portion of nozzles where subsonic flow exists; in the divergent portion of nozzles where supersonic flow is dominant; and externally in expanding supersonic flows around bodies such as ogive-cylinder and sphere-cylinder configurations. Some of the theoretical and experimental work is reported in Refs. 1-5. Many of these works are experimental in nature. Patel and Head in Ref. 1 have shown experimentally that quite large departures occur from the universal inner-law velocity distribution in the presence of severe favorable pressure gradients in turbulent boundary layers. The work of such investigators as Launder in Ref. 2 has described investigations generally similar to that reported by Patel et al. (Ref. 1), but emphasizes the measurements of turbulence and mean velocity proflles, and covers the complete reversal transition process. In the measurements of Back, Cuffel and Massler (Ref. 3), a reduction in heat-transfer below values typlcal of a turbulent boundary layer was found when the values of the parameter, $k=\left(\mu_{e} / \rho_{e} U_{e}\right)\left(d U_{e} / d x\right)$ exceeded about 2 to $3 \times 10^{6}$. One of the best documented experimental investigations of compressible boundary layer relaminarization is that reported by Nash-Webber (Ref. 4). In this work, an instrumented flat plate was tested in the presence of a variety of upper-wall profiles. The profiles were chosen to impose varlous pressure gradients on the flat-plate turbulent boundary layer. He deduced a comprehensive criterion for
relaminarization, which will be discussed in detall in the following subsection. It was noticed that acceleration effects tend to keep flow laminar beyond the normally-prescribed transition value.

### 3.2 Relaminarization Criterion

The various turbulence models in BLIMPJ were derived based on zero to moderate pressure gradients existing in the flow direction and thus, would not be able to predict laminarization for severe favorable pressure gradients. However, a treatment done by Adams et al. (Ref. 5) using the IKET (Integral form of the Kinetic Energy of Turbulence) approach was able to predict Iaminarization on the shoulder of a spherecylinder configuration tested at $M_{\infty}=9$ in Tunnel $F$ at AEDC. It was also pointed out by Adams that BLIMP could not predict either the onset of relaminarization or the degree of relaminarization.

The acceleration parameter which is a potential candidate for relaminarization and chosen for this study is that due to Nash-Webber (Ref. 4). According to Ref. 4, the acceleration parameter is defined as,

$$
\begin{equation*}
K=\frac{\bar{\mu}_{W}}{\bar{p}_{W} U_{e}} \cdot \frac{d U_{e}}{d x} \tag{3.1}
\end{equation*}
$$

Where the subscript ' $w$ ' denotes wall conditions, the subscript 'e' denotes boun-dary-layer edge conditions, and the barred quantities are time-averaged values. The importance of this parameter is lllustrated in Ref. 4 and is reproduced here in Fig. 3.1 for completeness. According to this, the numerical value of $K$ can be used as an indicator for probable occurrence of relaminarization provided that the momentum thickness Reynolds number based on edge conditions is sufficlently low. The recommended boundary value for the onset of relaminarization in FIg. 3.1 seems to be somewhat lower than the threshold recommended by Launder


Fig. 3.1 Turbulent-Laminar Transition Boundary
(Ref. 2) and was curve-fitted by a quadratic polynominal given by

$$
\begin{equation*}
K=a R^{2}+b R+c \tag{3.2}
\end{equation*}
$$

where

$$
\begin{aligned}
& a=8.935 \times 10^{-14} \\
& b=2.239 \times 10^{-10} \\
& c=1.0248 \times 10^{-6}
\end{aligned}
$$

The end of relaminarization (complete laminar condition) is the limit (Fig. 3.1) suggested by Kline (given in Ref. 4) where there is complete suppression of turbulence production.

Currently, BLIMPJ contains a criterion for transition where a specified or input value of $\operatorname{Re}_{e, \theta}$ is used to trigger transition. When the prescribed $\mathrm{Re}_{e, \theta}$ is exceeded, the turbulent transport properties are introduced into the calculations. In order to simulate a transition zone, these transport properties are reduced by a factor varying between 0 and 1 for complete laminar and complete turbulent flow respectively. A linear relationship that is used for varying $\varepsilon_{m}$ (eddy viscosity) is given by

$$
\begin{equation*}
\varepsilon_{m}=l(S) \cdot \varepsilon_{m}(\text { ref }) \tag{3.3}
\end{equation*}
$$

where $\varepsilon_{m}$ (ref) is the reference value for complete by turbulent flow and

$$
\begin{aligned}
& \\
\text { with }(S) & =\frac{s}{S_{+}}-1.0, S_{+}<S<2 S_{+} \\
I(S) & =0 \text { for } S \leq s_{+} \\
I(S) & =1 \text { for } S \geq 2 S_{+}
\end{aligned}
$$

where $S$ is the running length and $S_{t}$ is the running lenght up to the point of transition on the body. It is suggested by Ref. 6 that a flat plate zero pressure gradient value of $R_{e, \theta}=360$ serves as a nominal estimate. Now, In order to
account for flow acceleration effects, the recommended transition boundarles described in the previous paragraph and given in Fig. 3.1 has been coded in BLIMPJ. For acceleration parameters K less than $1 \times 10^{-6}$, Eq. (3.3) is used to check the state of the boundary layer. However, for acceleration parameter greater than $1 \times 10^{-6}$, the new criterion given in Fig. 3.1 and described in the previous paragraph is used. In order to simulate a relaminarization zone, the values of $K$ are used instead of $S$ in Eq. (3.3). $K_{1}$ and $K_{2}$ at any $\mathrm{Re}_{e, \theta}$ corresponding to the beginning and the end of relaminarization have been coded in BLIMPJ according to the following formula:

$$
\begin{equation*}
\varepsilon_{m}=\left(\frac{K-K_{1}}{K_{2}-K_{1}}\right) \cdot \varepsilon_{m}(r e f) \tag{3.4}
\end{equation*}
$$

It should be observed that $\varepsilon_{m}$ linearly varies with $K$ from a turbulent $\varepsilon_{m}$ (ref) value to a value of zero for completely laminar flow. Incidentally, the percent relaminarization value is

$$
\begin{equation*}
P C T=\left(\frac{K-K_{1}}{K_{2}-K_{1}}\right) \times 100 \tag{3.5}
\end{equation*}
$$

This additional logic in BLIMPJ only applies for turbulent flow. Depending on the value $K$, a value of turbulent eddy viscosity is calculated and fed into the boundary layer calculations.

### 3.3 Examples

In order to check the limits of relaminarization, an example of flow over the Shuttle clean ET configuration was considered. The aeroheating data were measured on a 0.0175 scale clean ET model tested at $M_{\infty}=7.3$ in the Ames HWT facility. The measured data had been compared against turbulent and laminar cal-
culations made by other aeroheating codes in Fig. 3.2. Because of tripping of the boundary layer due to the ET triple-cone nose, the boundary layer becomes turbulent over the ogive. The flow remains turbulent up to $X / L=0.2$, becomes fully laminar at $X / L=0.25$, and finally turbulent agaln beyond $X / L=0.4$. The acceleration parameter in Eq. (3.2) was examined after calculating the pressure gradient from the method-of-characteristics procedure and then the acceleration parameter, and was plotted in Fig. 3.3 as a function of $X / L$. It is evident that the parameter peaks at $X / L=0.2$ and drops of $f$ very rapidly as $X / L$ is increased. Another way of plotting this information is shown in Fig. 3.4, where K is plotted vs. Re $e, \theta$. From both the figures, it is obvious that the peak value is not higher than the threshold value of $K\left(=1.58 \times 10^{-6}\right)$ at $\operatorname{Re}_{e, \theta}=1550$. This indicates that the acceleration parameter is not high enough to trigger relaminarization, even though the data seem to suggest it. A similar observation was made by Adams (Ref. 5) for the sphere-cylinder case. Even though his IKET approach as well as the measured data seemed to show relaminarization, the Nash-Webber correlation did not strongly suggest that.

In order to examine the validity of this correlation for nozzle boundary layers, the relevant data taken on a $10^{\circ}-10^{\circ}$ half angle conical nozzle by Back et al. (Ref. 3) were examined in Fig. 3.5. Wall pressures calculated by TDK (Ref. 7) were input to the REMTECH version of BLIMPJ, and the heat-transfer. (Fig. 3.5.B) along with the acceleration parameter distributions (Fig. 3.5.C) were calculated. The acceleration parameter based on edge quantities, $K_{e}$, compared quite well with Back's calculations. The $K_{e}$ peak occuring upstream of the nozzle throat was not predicted by BLIMPJ because of inadequate wall pressure definition in this region. The heat-transfer calculations were made by using the coded relaminarization criterion. The momentum thickness Reynolds number, Re $_{e, \theta}$ distribution compared well with Back's calculations. The $K_{w}$
vs. $R_{e, \theta}$ correlation for relaminarization [Eq. (3.2)] suggested that the turbulent boundary layer was on the verge of relaminarization at the tangency point located at the juncture of the conical and curved portions of the nozzle contraction section. It is seen from Fig. 3.5.B, however, that the prediction is consistently higher than the measured data and that the boundary layer is predicted to be turbulent throughout the contraction section of the nozzle, but not partially relaminarized as evident from the measured data and as pointed out by Back's analysis. Back et al. point out in their paper that if $\mathrm{K}_{\mathrm{e}}$ is higher than 2 to $3 \times 10^{-6}$ relaminarization occurs. Since $K_{e}$ satisfies this criterion In the contraction portion of the nozzle as evident from Fig. 3.5.C, it suggests that relaminarization occurs. The currently coded criterion, which is different from the above criterion and is more definite in structure, is not able to quantify the degree of laminarization as well as suggested by Nash-Webber (Ref. 4). An example of the name list input to turn on the relaminarization flag is given in Table 3.1.

### 3.4 Dlscussions

The Nash-Webber criterion for relaminarization worked only marginally for the external flow situatlons, whereas for the limited measured data avallable on nozzles where relaminarization occurs in the boundary layer, this criterion seems to be only approximate. Without going through an extensive analysis such as the IKET-type model (Ref. 5), the current approach needs to be modified somewhat for engineering calculations. In addition, relaminarization can be predicted in the presence of roughness. In order to accomplish this, the roughness option 3 due to Cebecl must be input ( $\mathrm{RK}=\ldots .$. ICF $=3$ ) along with the relaminarization option (ILAMIN $=1$ ). The occurence of relaminarization will tend to reduce the turbulence length scales whereas the presence of wall rough-

```
FEMTEO\mapstoINO.
```



Fig. 3.2 1.75\% Model Space Shuttle External Tank Heat-Transfer Distribution.
$k \times 10^{6}$

$\mathrm{Re}_{\mathrm{e}, \theta}$

Fig. 3.3 Plot Of Acceleration Parameters Based On Edge And Wall Conditions And Momentum Thickness Reynolds Number Vs. X/L For The Shuttle ET Model


Fig. 3.4 Acceleration Parameter Vs. Re $e_{e, \theta}$ for the Shuttle ET Shoulder


Fig. 3.5 Relaminarization Analysis Of The Boundary Layer Flow In Back Et Al. $10^{\circ}$ - $10^{\circ} \mathrm{Half}$ Cone Angle Nozzle

TABLE 3.1 Example Of Namelist Input For Relaminarization

ness will tend to increase it. Although the code has not been exerclsed extensively for both being present in a nozzle, it is belleved that the code would handle it adequately.

### 3.5 References

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Section 4.0

## PARTICLE EFFECTS

### 4.1 Background

The study of the boundary layer flow contalning particles (in the fluld-particle systems) is of special interest because of the influence of the particles on the wall shear and heat transfer, the possible tendency of particles to collect near a wall, and the problem of particle impingement on the wall. Typical data (Ref. 1) in the chemical engineering ilterature correlated In terms of voidage show that there is negligible effect caused by solid particles until the volume percent of solids reaches 0.05 percent, but a very marked Increase occurred in heat transfer for higher solids loading. In fact, Nusselt number increases by factors as high as eight have been reported for the addition of particles to a flowing gas (Ref. 1). Materlal deposited on the nozzle wall al so represents a loss in performance, because the resulting rough surface causes increased skin friction losses.

Correlation of gas-particle heat transfer in terms of sollds loading and, sometimes, tube diameter (for pipe flow) is not entirely satisfactory, since such correlations ignore the effect of particle size. The differences in the data reported by Leva (Refs. 2 and 3) suggest that the enhancement in heat transfer is at least partially associated with disturbance of the laminar sublayer by particles, causing a local increase in heat transfer. On the other hand, reduction in heat transfer and shear stress have been reported in Ref. 4 for large populations of the smallest particles, less than $1 \mu$, by primarily displacing the boundary layer and thereby reducing thermal gradients.

The laminar particle-gas boundary layer has been investigated by Marble (Ref. 5), Soo (Ref. 6), Tabakoff and Hamed (Ref. 7) using momentum integral
techniques. In all these studies, analytical expressions have been found relating wall heat transfer and shear with particles to those without the particles. These investigations have determined that the introduction of particles leads to an Increase in the gas boundary layer thickness. In addition, it was found that the gas boundary layer characteristics are more sensitive to particle concentration than any other particulate flow parameter. It has been shown that for gas-particle flow systems, the wall heat transfer and skin friction are related to non-particle flow by a non-dimensional parameter called the "momentum range" which depends on particle size, the fluid viscosity, the fluid velocity and the distance from the leading edge, and another quantity called the "particle momentum interaction parameter", which depends on the ratio of particle mass density to fluid mass density.

Particulate-laden turbulent boundary layer flows in nozzles have not been understood completely and substantial empiricism must be employed to estimate the effects of particle concentration, particle size, density, pressure and entropy gradients on wall shear and heat transfer rate. Tien (Ref. 8) analyzed the increase in heat transfer due to differences in the gas and particle temperatures in boundary layer regions, under the assumptions of incompressible, constant property flow with no radiation or velocity lag effects and no effect of the particles on the gas flowfield. In this case, there is an increase in heat transfer rate while the flow is developing in the pipe. Soo and Tien (Ref. 9) considered particle motion in a turbulent fluid stream with emphasis on the effect of wall interference. The high particle intensity in wall regions increases the heat transfer by increasing the particle to gas heat transfer rates. Disruption of the gas laminar sublayer by particle motion further increases the local heat transfer. Also, if temperatures are high enough for radiation to occur, the radiation from particles to col der walls causes additional
heat transfer. Farbar and Morley (Ref. 10) also concluded from their experimental work on flowing gas-solids mixtures in a circular tube that the use of solids in gaseous heat transfer systems may prove to be advantageous when an increase in the heat transfer rate is desired without any increase in the heat transfer area. It was concluded from this study that the gas-side heat transfer factor increases rapidly for sollds loading ratios greater than unity. The solIds affect both the gas boundary layer and the heat capacity of the flowing mixture. On the other hand, for solids loading ratios of unity or less, a transitional region exists in which the effect is primarily one of increased heat capacity.

### 4.2 Particle eptions

The various options integrated in BLIMPJ fall into the following two categories:

### 4.2.1 Laminar Boundary Laver-Particulate Flow

The approach used in the modification of BLIMPJ to account for the presence of particles and their effect on wall shear and heat transfer is taken from the work of Marble (Ref. 10). Marble developed an expression for the shear coefficlent from an Integral momemtum solution of the laminar boundary layer equations, particle continuity and momentum equations for an incompressible flat plate flow. The final expression for the case where $\lambda_{v} / x<1$ is given in Table 4.1. The applicable momentum range, $\lambda_{v}$, in the OTV-type nozzles would fall basically in this category. We recognize in Eq. (B.1) of Table $4.1 \mathrm{C}_{\mathrm{f}_{0}}$ as the shear coefficient for the fluid boundary layer without particles. In his original paper, Marble used

$$
\begin{equation*}
C_{f_{0}} / 2=0.332 / \sqrt{R_{x}} \tag{4.1}
\end{equation*}
$$

Since BLIMPJ provides a shear coefficient for clean flow, that value was used as reference instead to calculate the shear coefficient for the gas-particle system. The quantity, $\lambda_{v}$, represents a distance, $x$, which describes the particle motion relative to the fluld. For $x<\lambda_{v}$, there is a high degree of fluid-particle silp, whereas for $x>\lambda_{v}$, the particles tend to take on the motions of the gas. The heat transfer characteristics are more complex in the high "particle-slip" regime in that the initial conditions become quite important in such a calculation. Since there is very litter work in the literature for this regime, this was not coded in BLIMPJ.

Returning our attention to the expression for shear, the factor $\sqrt{1+K}$ multiplying the usual shear coefficient gives the result for no particle slip and represents a minimum value for shearing stress. The first order correction $0.49\left(\lambda_{v} / x . K / 1+K\right)$ gives shear stress due to particle slip reduction along the flow path.

Heat transfer through the boundary layer was treated in a similar manner as given in Eq. (B.2).

### 4.2.2 Iurbulent Boundary Laver-Particulate Flow

The approach for modification of the heat transfer and skin friction calculations in BLIMPJ for a turbulent boundary layer is based on the analytical results of Tlen (Ref. 8) and the empirical expressions of Farbar and Morley (Ref. 10). Tien solved the turbulent gas-particle energy equations for flow in a plpe and found that the qualltative effect of particle concentration is to flatten the temperature profile and consequently to increase the heat transfer. He has theoretically confirmed the test results of Farbar and Morley that

TABLE 4.1

## GAS-PARTICLE SKIN FRICTION AND HEAT TRANSFER

## Laminar Boundary Layer (Marble)

$$
\begin{equation*}
c_{f}=c_{f_{0}} \sqrt{1+K}\left(1+0.49 \frac{K \lambda_{v / x}}{1+K}\right), \frac{\lambda_{v}}{X} \ll 1 \tag{B.1}
\end{equation*}
$$

and

$$
\begin{equation*}
\dot{q}=\dot{q}_{0} \sqrt{1+k}\left(1+0.49 \frac{k \lambda_{v / x}}{1+k}\right), \frac{\lambda_{v}}{x} \ll 1 \tag{B.2}
\end{equation*}
$$

where

$$
\begin{aligned}
K & =\rho_{p} / \rho_{e} \\
\lambda_{v} & =\frac{m U_{e}}{6 \pi a \mu_{e}}
\end{aligned}
$$

## TABLE 4.1 (Continued)

## Turbulent Boundary Layer

For
$C_{f}=C_{f_{0}}\left(1+\beta_{5}\right)$
and $\quad \dot{q}=\dot{q}_{0}\left(1+\beta_{5}\right)$
(B.3)
where $\quad \beta_{5}=\frac{C_{p} W_{p}}{C_{f} W_{f}}$

For

$$
\begin{align*}
& \frac{W_{p}}{W_{f}}>1 \text { (Farbar and Morley) } \\
& N u=0.14 \operatorname{Re}_{D}^{0.6}\left(W_{p} / W_{f}\right)^{0.45}  \tag{B.4}\\
& \dot{q}=\frac{N u \cdot K_{g}}{D} \cdot\left(T_{a w}-T_{W}\right) \\
& \text { Particle Factor }=\dot{q} / \dot{q}_{o} \\
& C_{f}=\left(\frac{\dot{q}}{\dot{q}_{o}}\right) \cdot C_{f o} \tag{B.6}
\end{align*}
$$

## Nomenclature


suspended solids, having a solids-to-gas loading ratlo of less than 1.0 , have a negligible effect on heat transfer. As pointed out earlier. Tien's analysis is valid for the entrance region of a pipe. Since the flow is not fully developed in this region of the plpe, the boundary layers do not merge. This flow situation is similar to what happens in a nozzle, where the boundary layers develop near the nozzle wall and do not merge. Consequently, the expressions developed by Tien for the pipe may be applicable to a nozzle. The expressions for parti-cle-to-fluid loading ratlo of less than 1 are given in Eq. (B.3) of Table 4.1.

For higher particulate loading where interactions and collisions among particles become important, the above expression is no longer valid. For the case, where the particle-to-fluid loading ratio is more than 1 , the experimental results of Farbar and Morley (Ref. 10) have been correlated and are given in Eq. (B.4) of Table 4.1. This expression is valld for a limited Reynol ds number range of $13,500<R_{e}<27,000$ which were the limits in the test conditions. It has further been noted by Farbar and Morley that for loading ratlos up to unity, the Nusselt number varies as the 0.03 power of the loading ratio, while that above unity varies as the 0.45 power of the loading ratio, except that for the lowest Reynolds number which Indicates a variation to the 0.5 power. The expressions for Nu in Eq. (B.4) was used to calculate a particle factor which was then used to calculate skin friction coefficient from Eq. (B.5). The above expressions were coded in BLIMPJ and checked with a few examples.

### 4.3 EXAMPLES

In order to illustrate the effect of particles in the fluid boundary layer on skin friction and heat transfer rate, the following hypothetical example was chosen. Aluminum particles of 10 radius (density of $A 1=169 \mathrm{lbm} / \mathrm{ft}^{3}$ ) and par-ticles-to-fluid loading ratio of 0.5 was chosen. Thus,

$$
\begin{aligned}
r & =10 \mu=10^{-5} \mathrm{~m} \\
\rho_{\mathrm{al}} & =169 \mathrm{lbm} / \mathrm{ft}^{3} \\
C_{\mathrm{p}_{\mathrm{al}}} & =0.208 \mathrm{Btu} / \mathrm{lbm} \cdot{ }^{\circ} \mathrm{F}
\end{aligned}
$$

An example of the namelist input in BLIMPJ for particles-in-flow is given in Table 4.2. The OTV nozzle was used for testing the effects of these particles. The relative magnitudes of the resultant skin friction and heat flux are plotted in Figs. 4.1-4.3. Since the OTV nozzle contains both laminar and turbum lent boundary layer flow regimes, both laminar and turbulent expressions for particles-in-flow could be checked out simultaneously.

### 4.4 Discussions

The particle options chosen in the present work are designed to perform "point" calculations and are not capable of taking into account the "history" effects. The particle option can either be used independently or used along with one or both of the roughness and relaminarization options. The reference value for the particle factor will be obtalned either from the smooth wall value or from the relaminarlzation or rough wall value and then be enhanced by the particle factor. It has been polnted out previously that the particle factor expressions for turbulent flow were derived from tube data and do not represent a rocket nozzle case, and in that sense are only approximate in nature. However, they wlll provide relative values of wall skin friction and heat flux for varlous particle sizes and particle loadings. Some relevant suggestions for future work for gas-particle flows in rocket nozzles are given in Sec. 6.

TABLE 4.2 Example Of Namelist Input Particle Option


Fig. 4.1 Comparison Of Heat Flux Distribution Over The OTV Nozzle Wall For With And Without Particles In Flow


Fig. 4.2 Comparison Of Stanton Number Distribution Over The OTV Nozzle Wall For With And Without Particles In Flow


Fig. 4.3 Comparison Of Skin Friction Coefficient Distribution Over The OTV Nozzle Wall For With And Without Particles In Flow

### 4.5 References

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Section 5.0
THRUST LOSS RE-EVALUATION

### 5.1 Background

A thrust loss calculation method which has been prevlously implemented in BLIMPJ code is given in Ref. 1. The thrust loss due to the boundary layer ef-: fects for a circular cross-section nozzle is given at a specified cross-section by (for vacuum ambient conditions)

$$
\begin{equation*}
\Delta F=2 \pi r_{e} \cosh _{e}\left(\rho_{e} U_{e}^{2} \theta-P \delta_{B}^{*}\right) \tag{5.1}
\end{equation*}
$$

where

$$
\begin{aligned}
& r_{e}=\text { Body radius at the station of interest } \\
& \phi_{e}=\text { Wall angle } \\
& \rho_{e}=\text { Boundary layer edge density } \\
& U_{e}=\text { Boundary layer edge velocity } \\
& \theta=\text { Momentum thickness } \\
& P=\text { Static pressure in the boundary layer } \\
& \delta_{B}^{*}=\text { Body displacement thickness }
\end{aligned}
$$

The assumptions used in deriving the above expression are the following:
(i) The boundary layer is thin, l.e., the thickness of the boundary layer is small compared to the radius of the nozzle at any cross-section.
(Ii) The inviscid values of density and velocity do not change within the thickness of the boundary layer. In other words, if there was no viscosity (l.e. for inviscid flow), there would be no variation of the inviscld values between the edge location and the nozzle wall.
(ili) The pressure is constant across the boundary layer. This assumption is consistent with the derivation of the usual boundary layer equations.
(iv) The definitions of body displacement thickness and momentum thickness are given by

$$
\begin{equation*}
\delta_{B}^{*}=\int_{0}^{e}\left(1-\frac{\rho U}{\rho_{e} U_{e}}\right) d y \tag{5.2}
\end{equation*}
$$

and

$$
\begin{equation*}
\theta=\int_{0}^{e} \frac{\rho U}{P_{e} U_{e}}\left(1-\frac{U}{U_{e}}\right) d y \tag{5.3}
\end{equation*}
$$

where e and o refer to edge and wall conditions respectively.
As the nozzles grow in area ratio, the boundary layers grow in size, and the above assumptions may not hold. The proposed OTV nozzles such as the one given in Fig. 2.5 will utilize an expander cycle operations mode in which the walls will be regenerately cooled and the heat energy will be used to drive the turbines and pumps. So, while the regenratively cooled walls wlll help in reducling the size of the boundary layers to some extent, the large area ratio nozzles wlll produce thick boundary layers. Consequently, depending on the reservolr and exit conditions, and the geometry of the nozzle, it is possible and very likely that boundary layer thicknesses wlll vary from small to large values. The displacement and transverse curvature effects become important for thick boundary layers and must be included in the boundary layer calculations. In addition, as the flow expands in the nozzle, it will create low density and high Mach number flows. If the flow passes from the continuum to a non-continuum regime, velocity slip and temperature jump (STJ) may become important.

Similar boundary layer solutions are not applicable for such an Investigation, since similarity cannot be satisfled for any specified set of reservoir conditions, nozzle geometry and wall temperature distributions. Fortunately, the boundary layer procedure in BLIMPJ does not assume similarity. Furthermore, It takes into account transverse curvature effects (TVC) in the derivation. It also calculates the displacement effects for thin boundary layers. As far as the STJ effects are concerned, it has been pointed out by previous investiga-
tions (Ref. 2) that they are generally small compared to the other effects discussed above and thus, will be ignored in the present approach.

### 5.2 Thrust Loss Reevaluation Procedure For Thick Boundary Layers

In accordance with the above discussions, the expression for thrust loss for thick boundary layers has been modified. The assumptions made in deriving Eq. (5.2) and (5.3) are no more strictly valid. The u-component of the velocity In the inviscid flow will vary to some extent between the nozzle wall to the edge location. Consequently, the definitions for $\delta_{B}^{*}$ and $\theta$ are

$$
\begin{equation*}
\delta_{B t}^{*}=\int_{0}^{e}\left[1-\frac{\rho U}{\rho_{i}(y) \cdot U_{i}(y)}\right] d y \tag{5.4}
\end{equation*}
$$

and

$$
\begin{equation*}
{ }_{t}^{\theta}=\int_{0}^{e} \frac{\rho U}{\rho_{1}(y) \cdot U_{1}(y)}\left[1-\frac{U}{U_{i}(y)}\right] d y \tag{5.5}
\end{equation*}
$$

The expression for the thrust loss calculation given in Eq. (5.1) will al so have to be modifled in its derivation where the edge quantities, ( $\rho_{e}, U_{e}$ ) and pressure will no more be constants but would be replaced by local inviscid values $\rho_{i}(y), U_{1}(y)$ and $P(y)$. However, it was decided that the whole procedure of thrust loss calculation will be much more simple and adapted a lot easier in the BLIMPJ algorithm, if the pressure is replaced by an average value of the pressure distribution within the thlckness of the boundary layer. As a result, averaged inviscid edge values of velocity and density wIll automatically be calculated from the BLIMPJ algorithm and could be used in the thrust loss calculation in the existing algor ithm in BLIMPJ. In the above calculations, the location of the boundary layer edge is not precisely known and has to be determined by lterating upon the inviscid and viscous flowflelds.

There are two different problems to be solved when one attempts to calcu-
late performance for a rocket nozzle having thick boundary layers:
Case 1 - The potential nozzle contour is given and the objective is to define the hardware wall contour and calculate the rocket nozzle performance. For detalls, see Attachment 5.1.

Case 2 - The hardware wall contour is given and the objective is to define the potential contour and calculate the rocket nozzle performance. For detalls, see Attachment 5.2.

Attachment 5.1

In the case, where the potential contour is given, the objective is to define the wall contour for thick boundary layer situations. The suggested iteration procedure is given below: (Also see Fig. 5.1).
(1) Run the inviscid code (TDK and RAMP) to define the distribution of pressure on the potential wall and everywhere else in the nozzle, particularly near the potentlal wall.
(ii) Run BLIMPJ with the giken pressure distribution on the potential wall. This calculates $\delta$ and $\delta$. Then, the body radius is calculated from

$$
R_{B}=R_{P}+\delta^{*} \cos \phi
$$

This is the first iteration.
(ili) Calculate an average inviscid pressure for the height between the potential wall and the boundary layer edge, which was obtalned from the previous calculation at each station. Use these pressures to run BLIMPJ again, and calculate $\delta_{2}$ and $\delta_{2^{*}}^{*}$. Then, calculate $R_{B}$. This is the second iteration.
(iv) Iterations stop when convergence on $\delta^{*}$ is achleved within a specified accuracy.

Attachment 5.2

In the case where the hardware wall contour is given, the objective is to define the inviscid edge for thick boundary layer situations. The suggested Iteration procedure is given below: (also see Fig. 5.2)
(1) Run the inviscid code (TDK or RAMP) to calculate the distribution of pressure on the hardware wall in the nozzle.
(II) Run BLIMPJ with the calculated wall pressure distribution on the hardware wall. This calculates $\delta$ and $\delta$ as a function of the nozzle axial coordinate. Then, the radius of the potential wall is calculated from

$$
R_{P}=R_{B}-\delta^{*} \cos \phi
$$

This is the first itertation.
(ili) Calculate the pressures again by using the inviscid code (TDK or RAMP) on the new potential wall and everywhere else in the nozzle, particularly near the potential wall.
(iv) Calculate the average pressure for the height between the boundary layer edge, which was obtained previously, and the hardware wall. Use the pressures on the hardware wall to run BLIMPJ again and calculate $\delta_{2}$ and $\delta_{2}$. Then calculate $R_{p}$. This is the second iteration.
(v) Ge back to (III) and iterate until a prescribed convergence criterion on $\delta^{*}$ is achleved. If it is found that the pressure calculations in (iil) in the first two iterations are very close, do not go back to (ill), instead go back to the beginning of (iv).

Once the iterations are completed, the thrust loss wlll automatically have been calculated by BLIMPJ to yield the final answer.

## NOTES:

1. The potential contour is given; the objective is to define the wall contour.
2. Subscript refers to iteration number.
3. $\delta^{*}$ refers to displacement thickness.
4. $\delta$ refers to boundary layer thickness.

PPotential Contour


Fig. 5.1 Suggested Iteration Procedure For Nozzles With Thick Boundary Layer (Potential Contour Given)

NOTES:

1. The wall contour is given; the objective is to define the inviscid edge.
2. Subscript refers to iteration number.
3. $\delta_{*}$ refers to displacement thickness.
4. $\delta$ refers to boundary layer thickness.


Sample Of Iterated Contours
Fig. 5.2 Suggested Iteration Procedure For Nozzle With Thick Boundary Layer (Wall Contour Given)

### 5.3 Example

For lllustrating the procedure given above for calculating thrust loss for thick boundary layer situations, the OTV nozzle given earlier in Sec. 2.3 was used. Furthermore, since the glven wall coordinates represent a generic class of OTV nozzles, these coordinates were assumed to represent the potential wall contour of the OTV nozzle. Consequently, the iterations were performed based on the procedure shown in Attachment 5.1.

SInce REMTECH did not have the information to run TDK for computing and storing the pressures for the interior points away from the wall, another available code called RAMP (Ref. 3) was run for the OTV nozzle contour to compute the pressure fields both on the wall and near the wall. Figure 5.3 gives a comparlson of wall pressure distributions from TDK and RAMP on the nozzle wall. The comparison is quite good. A comparison of $\delta^{*}$ calculations based on results from both codes is given in Fig. 5.4 showing a close agreement. The pressure distribution near the potentlal contour obtalned from RAMP is given in Fig. 5.5 along with $\delta$ and $\delta^{*}$ from the first iteration. It is obvious that there is a distribution of pressure through the thickness of the boundary layer and as a result, the shown inviscid edge of the boundary layer is not accurate. Golng through the step (ili) in Attachment 5.1 ylelds a new average pressure distribution given in Fig. 5.6, which is distinctly different from the first iteration both in the high pressure region near the throat and in the low pressure region near the exit plane. The BLIMPJ calculation yielded a $\delta^{*}$ distribution which was compared with the original distribution in Fig. 5.7. Again, the two iterations are somewhat different. A third iteration was done when it was found that the average pressure and $\delta^{*}$ distributions were very close to the second iteration (Figs. 5.6 and 5.7). The thrust loss in the successive iterations is given in FIg. 5.8.


Fig. 5.3 Comparison Of OTV Nozzle Wall Pressure Distribution Using Two Different Codes

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FEMTECHINC.
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Fig. 5.4 Comparison Between TDK And RAMP Output For The Boundary Layer Effective Displacement For First Iteration


Fig. 5.5 Inviscid Pressure Distribution At Various Wall Locations Within The Boundary Layer Thickness


Fig. 5.6 Iterations Of The Wall Pressure Distribution For The OTV Nozzle With Thick Boundary Layer


Fig. 5.7 Iterations Of $\delta^{*}$ Distribution For The OTV Nozzle With Thick Boundary Layer


Fig. 5.8 Thrust Loss In The OTV Nozzle As A Function Of Iterations

### 5.4 Discussions

The procedures described before and the example given in Subsection 5.3 are engineering procedures which could be used for thrust loss calculation in nozzles with thick boundary layers. The calculations performed at the time were not all computerized and as a result, could contain some inaccuracies in the varlous steps of the calculation. Even though the convergence was observed in the pressure distribution in Fig. 5.6 in the third iteration, it was not absolutely so in the convergence of $\delta^{*}$ in Fig. 5.7 and thrust loss in Fig. 5.8. However, the difference between the second and third iteration for the thrust loss in the OTV nozzle is around 10 ibs and it might be even less between the third and a fourth iteration. The thrust loss for thick boundary layers has not been programmed, since TDK cannot presently provide the necessary data away from the nozzle wall. However, a number of suggestions are made in Sec. 6 for future work.

### 5.5 References

1. Evans, M., "BLIMPJ User's Manual," Aerotherm/Acurex Corporaton, July 1975, under Contract NAS8-30930.
2. Whitfield, D.L., and Lewis, C.H., "Boundary-Layer Analysis of Low-Density Nozzles, Including Displacement, Slip, and Transverse Curvature," Journal of Spacecraft, April 1970, pp. 462-468.
3. Smith, Sheldon D., MHigh Altitude Chemically Reacting Gas-Particle Mixture, Vol ume 1 - A Theoretical Analysis and Development of Numerical Solution," August 1984, LMSC-HREC TRD867400-1.

Future work in the OTV research and development areas described in the previous sections may be categorized into three broad areas;

- Analytical
- Numer Ical
- Experimental


### 6.1 Analytical

The future analytical work on OTV-class nozzles, with reference to the four modules that have been addressed in the previous sections of this report, consists of the following recommendations:

### 6.1.1 Wall Roughness Effects

1. Roughness module in BLIMPJ needs to be checked out further with other avallable data for any size nozzle. This would enhance confidence in the usability of the various roughness options incorporated in BLIMPJ. The modules should also be exercised with the data to be taken on the future OTV model or flight tests.
2. Effects of partially smooth and partlally rough nozzle wall on wall skin friction and heat transfer rate need to be examined. This problem does not lend itself to the assumption of an equivalent sand roughness, because the concept of equivalent sand roughness which is based on simllarlty assumptions breaks down. Some related developments appear in works by T.C. LIn, J.C. Adams, etc.

### 6.1.2 Relaminarization

1. The relaminarization module needs to be checked out with any other avallable data for internal flow situations.
2. Questions remain as to whether the relaminarization criterion using wall quantities rather than edge quantities is valid for OTV-type nozzles. What happens to this criterion when wall roughness is present?
3. It is well known that freestream turbulence is present in the inviscid flow inside the nozzle. The question, then, is what role does the freestream turbulence play in the turbulence length scales and thus, in the relaminarization process?

### 6.1.3 Partlcle Effects

1. Check the options in BLIMPJ with avallable data both in laminar and turbulent flows.
2. For the case of replaceable and ablating nozzle inserts, the particles or debris in boundary layer flow wlll enhance heat transfer at the nozzle wall. If the particle loading could be determined, the effects of ablating nozzle wall could be determined.
3. Modify the turbulent mixing length due to the presence of particles in the flow.

### 6.1.4 Thrust Loss Reevaluation

1. Check the predicted performance with avallable nozzle data having large area ratlos, and consequently, thick boundary layers.
2. A procedure which consists of a combination of machine and hand calculations has been given in Sec. 5 for computing final performance calculations for large area ratio nozzles. This procedure should be considered approximate. A special software using a flow diagram involving TDK and BLIMPJ needs to be written for smooth calculation of high area ratio nozzles.
3. An optimization procedure needs to be developed to design a nozzle with length and area ratio constralnts for minimizing thrust loss in large area ratio nozzles.

### 6.2 Numerical

Computational fluid dynamics (CFD) procedures should be examined to evaluate the nozzle wall thermal losses due to relaminarization, the presence of wall roughness and particles in flow. Without going Into too many detalls, the following concerns should be borne in mind:

1. The turbulence models in the existing codes need to be examined. The problems of modifying the turbulence models for roughness, particles and relaminarization remain.
2. Acceptable chemistry packages have to be integrated in the CFD codes.
3. On the positive side, the iteration procedure necessary for calculating the thrust loss for thick boundary layers is eliminated in the CFD procedure, since the code defines both the inviscid and viscous flowflelds in the nozzle at the same time. However, the thrust loss formula for nozzles needs to be integrated with the CFD code, if the boundary layer effects need to be singled-out.

### 6.3 Experimental

It is the opinion of the authors that not enough applicable experimental data is avallable for the OTV-class nozzles. In order to validate the modules described in this report, measurements need to be made to support them. The parameters that need to be measured, the size of the models, the kind of flow to be tested and the accuracles involved in conducting these tests are the items. described in modular form in Table 6.1. This table presents a number of choices and possibilities from which any combinations could be selected for future experimental programs to support the OTV nozzle development.

TABLE 6.1 Recormmendations For OTV Experimental Programs

| STUDY ITEMS <br> ITEMS TO EXPLORE | WALL ROUGHNESS EFFECTS |  | RELAMINARIZATION EFFECTS |  | PARTICLE EffECTS |  | THICK BOUNDARY <br> LAYER ISP LOSSES |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| EXISTING <br> DATA BASE <br> FOR NOZZLES | NO OTV NOZZLE DATA <br> - rocketdyne 4ok subscale chamber test AT MSFC <br> - for sSme, no internal nozzle data |  | BACK AND CUFFEL $10^{\circ}-10^{\circ} \mathrm{HALF}$ ANGLE CONE DATA NASH-WEBBER VARIABLE NOZZLE WHICH STUDIED RELAMINARIZATION EFFECTS | - | NO OTV NOZZLE OR ANY OTHER NOZZLE BOUNDARY LAYER DATA the avallable data base is FOR TUBES AND PIPES | - | NO OTV THICK BOUNDARY LAYER DATA |
| $\begin{aligned} & \text { MODEL TESTS - } \\ & \text { SHORT DURATION } \end{aligned}$ | - steady state test times 10 msec 100 MSEC <br> - test time dependent on altitude CHAMBER SIZE AND/OR DIFFUSER CAPACITY USE DIFFERENT NOZZLES OR NOZZLE INSERTS FOR ROUGHNESS EFFECTS STUDY | - | TEST ARRANGEMENT <br> SAME AS WITH <br> WALL ROUGHNESS |  | VERY DIFFICULT IF NOT IMPOSSIBLE TO INJECT KNOWN PARTICLES INTO FLOWS ON SHORT DURATION BASIS |  | $\cdots$ |
| COLD, HOT OR REACTIVE FLOW | - EXACT SIMULATION OF HOT FLOWING $\mathrm{H}_{2} / \mathrm{O}_{2}$ C $0 / F=6$ AND $P_{C H}=2000$ PSI <br> USE OF COLD/NON-REACTING GASES | - | SAME AS WITH WALL ROUGHNESS | - | COMBUSTION OF SOLID <br> PROPELLANT - THE PROBLEM IS THE LACK OF CONTROL OR KNOWLEDGE OF PARTICLE SIZE AND CONCENTRATION |  |  |
| PARAMETERS TO BE MEASURED | - wall roughness <br> - nozzle wall heat transfer as a FUNCTION OF TIME <br> - NOZZLE WALL PRESSURES <br> - NOZZLE WALL TEMPERATURES <br> - EXIT VELOCITY/TEMPERATURE PROFILES |  | WALL HEAT TRANSFER <br> WALL TEMPERATURES <br> WALL PRESSURES | 0 | WALL HEAT TRANSFER <br> WALL TEMPERATURES <br> WALL PRESSURES |  |  |
| INSTRUMENTS <br> TO BE USED; SPECIAL innovative PROBES THAT COULD BE USED | - FAST-RESPONSE PIEZO-ELECTRIC PRESSURE TRANSDUCERS <br> - THIN FILM SHORT DURATION HEAT TRANSFER GAGES <br> CO-AXIAL SURFACE HEAT TRANSFER GAGES MINIATURE THIN WIRE/T.C. GAGES mechanical measurements of mall roughness | - | SAME AS WITH WALL ROUGHNESS | - | same as mith mall roughness |  |  |
| ACCURACY OF MEASUREMENTS | - $\pm 5 \$$ TO $\pm 10 \$$ FOR THIN-FILM AND CO-AXIAL GAGES <br> - $\pm 0.58$ ON TEMPERATURE <br> - $\pm 2 \%$ ON PRESSURE |  | $\pm 5 \$$ TO $\pm 10 \$$ ON HEAT TRANSFER $\pm 0.5 \%$ ON TEMPERATURE $\pm 28$ ON PRESSURE | - | $\pm 5 \$$ TO $10 \%$ ON HEAT TRANSFER $\pm 0.5 \%$ ON TEMPERATURE $\pm 28$ ON PRESSURE |  |  |
| MODEL SCALE PROBLEMS, IF ANY | - CHEMISTRY, THERMODYNAMICS AND TRANSPORT PROPERTIES ARE REALISTIC IN NOZZLE <br> - SMALL THROAT AREAS AND IMPERFECTIONS MAY OBSCURE EFFECTS BEING SOUGHT | - | SAME AS WITH MALL ROUGHNESS | - | Particle size and CONCENTRATIONS VERY DIFFICULT to scale for small test rig |  |  |
| FACILITIES TO BE USED | IMPULSE BASE FLOW FACILITY (IBFF) AT MSFC <br> - PLUMBROOK SPACE POMER FACILITY AT NASA LEWIS <br> - chamber a at johnson space center <br> - Ludwig tube at calspan, buffalo | - | SAME AS FOR WALL ROUGHNESS EFFECTS | - | SAME FACILITIES AS FOR WALL ROUGHNESS STUDIES |  |  |

TABLE 6.1 (Continued)

| STUDY ITEMS <br> ITEMS TO EXPLORE |  | WALL ROUGHNESS EFFECTS |  | RELAMINARIZATION EFFECTS |  | PARTICLE EFFECTS |  | THICK BOUNDARY LAYER ISP LOSSES |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| MODEL TESTS LONG DURATION | - | DEPENDING ON MODEL SIZE AND TEST DURATION, COSTS CAN BE A FACTOR OF 10 LARGER THAN SHORT DURATION <br> allows more than one measurement per run scale problems are alleviated <br> HIGH ALTITUDE SIMULATION REQUIRES VERY LARGE FACILITY |  | SAME AS FOR WALL ROUGHNESS EFFECTS | $\bullet$ | LONG DURATION ALLOWS FOR pOSSIBLE UTILIZATION OF PARTICLE INJECTION TECHNIQUES in COLD/WARM GAS FLOW reactive flows still have unknown PARTICLE SIZE/CONCENTRATION | - | LARGE OTV MODELS SHOULD BE USED <br> CAN MAKE BETTER THRUST MEASUREMENTS THAN SHORT DURATION TEST <br> BOUNDARY LAYER PROBING IS POSSIBLE |
| COLD, HOT OR REACTIVE FLOW | - | COLD/WARM FLOWS SIMPLEST AND LEAST COSTLY HOT OR REACTIVE FLOWS REQUIRE COMPLEX FACILITY AND MODEL COOLING |  | SAME AS FOR WALL ROUGHNESS EFFECTS | - | COLD, HOT OR REACTIVE FLOW SIMULATION IS NOW COMPLICATED BY THE NEED FOR PARTICLES PARTICLE INJECTION SCHEME CAN BE UNRELIABLE | - | hot/REACTIVE FLOWS SIMULATING H $\mathrm{H}_{2} / \mathrm{O}_{2}$ SYSTEM are preferabte |
| parameters TO BE MEASURED | - | WALL ROUGHNESS <br> WALL PRESSURE AND TEMPERATURE MEASUREMENTS <br> EXIT VELOCITY/TEMPERATURE PROFILES <br> WALL HEAT TRANSFER RATE <br> PROBE THE BOUNDARY LAYER INSIDE NOZZLE | $\bullet$ | SAME AS FOR WALL ROUGHNESS EFFECTS | - | PARTICLE DENSITY AND SIZE NOZZLE PROBES TO MEASURE PRESSURE AND TEMPERATURE WALL PRESSURE AND TEMPERATURE MEASUREMENTS | $\bullet$ | THRUST MEASUREMENT <br> BOUNDARY LAYER PRESSURE AND TEMPERATURE MEASUREMENTS. BOTH INSIDE AND AT EXIT PLANE OF NOZZLE |
| INSTRUMENTS AND PROBES TO BE USED | - | THERMOCOUPLES FOR TEMPERATURE PRESSURE TANSOUCERS <br> LASER DOPPLER VELOCIMETER (PARTICLES) FOR HOT/REACTIVE FLOW MEASUREMENTS, SYSTEMS/PROBES REQUIRE SPECIAL PROTECTION optical schlieren at exit plane PHASE CHANGE PAINT | $\bullet$ | SAME AS FOR WALL ROUGHNESS EFFECTS | $\bullet$ | LDV VERY ADAPTABLE TO PARTICLE FLOWS <br> PARTICLE MEASUREMENT TECHNIQUES ARE GENERALLY UNREL IABLE EXCEPT IN SPECIAL FLOW SITUATIONS | $\bullet$ $\bullet$ $\bullet$ | THRUST/STRAIN GAGE <br> - MEASUREMENTS FOR ISP DETERMINATION <br> intrusive techniques SUCH AS HOT-WIRE ANENOMETERS AND PRESSURE PROBES <br> NON-INTRUSIVE TECHNIQUE SUCH AS LDV |
| ACCURACY OF MEASUREMENTS | $\bullet$ | SAME AS FOR SHORT DURATION LDV $\pm 15 \%$ DUE TO PARTICLE LAG | $\bullet$ | SAME AS FOR SHORT DURATION | - | SAME AS FOR SHORT DURATION LDV $\pm 15 \%$ DUE TO PARTICLE LAG PARTICLE - UNKNOWN | $\bullet$ | SAME AS FOR SHORT DURATION HOT WIRE $\pm 15 \%$ |
| MODEL SCALE PROBLEMS, IF ANY | - | scale problems are alleviated to some EXTENT ASSUMING THAT MODELS ARE LARGER ON LONG DURATION <br> THROAT MUST BE PROTECTED AGAINST HIGH q RESULTING IN WALL TEMPERATURE discontinuity where materials change | - | SCALE PROBLEMS are alleyiated if MODEL SIZES ARE I NCREASED | - | scale problems are alleviated if MODEL SIZES ARE INCREASED | - | scale problems are alleviated if model SIZES ARE INCREASED |
| FACILITIES TO BE USED | - | engine test facility at aedc can SImulate altitude <br> ENGINE TEST FACILITY AT MSFC has NO ALTITUDE SIMULATION <br> LEWIS TEST FACILITY ALTITUDE SIMULATION NOT KNOWN |  | SAME FACILITIES AS FOR WALL ROUGHNESS STUDIES | - | SAME FACILITIES AS FOR MALL ROUGHNESS STUDIES | - | SAME FACILITIES AS FOR WALL ROUGHNESS STUDIES |

## APPENDIX

## A LISTING OF THE UPDATED SUBROUTINES IN BLIMPJ

| 0001 | 000565 | 4才1G | 0001 |  | 001023 | 42L | 0001 |  | 001042 | 447G | 0001 |  | 001057 | 452G | 0001 | 001072 | 464G |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0001 | 001107 | 500G | 0001 |  | 001124 | 513G | 0001 |  | 001131 | 520G | 0001 |  | 001172 | 533G | 0001 | 001246 | 65L |
| 0001 | 001252 | 70L | 0006 | 1 | 000000 | ADARY | 0005 | I | 000000 | CASE | 0005 |  | 000015 | CBAR | 0004 | 000000 | DUB |
| 0006 I | 000026 | ESPER | 0006 | I | 000027 | FLDP | 0007 | I | 000000 | FLDX | 0006 | I | 000313 | FLDY | 0000 | I 000000 | I |
| 0006 I | 1001077 | IAXIS | 0000 | 1 | 000003 | IERR | 0006 | 1 | 001100 | INJ | 0000 |  | 000071 | INUP\$ | 0006 | I 001101 | INKI |
| 0003 | 000000 | IPLOT | 0007 | I | 000030 | IPP | 0005 | I | 000020 | IS | 0005 |  | 000021 | ISH | 0003 | I 000001 | IUNIT |
| 0006 I | 001102 | IWALL | 0000 | 1 | 000001 | IX | 0006 | 1 | 001103 | J | 0006 | I | 001104 | K | 0006 | I 001105 | KELVIN |
| 00061 | 001106 | KGDE | 0006 | I | 001107 | L | 0006 | I | 026161 | LA | 0006 | I | 001110 | LANK | 0006 | I 026162 | LB |
| 0006 I | 026163 | LC | 0006 | 1 | 001111 | LOGRAM | 0006 | I | 001112 | LWXO | 0006 | I | 001126 | LWYO | 0006 | I 001142 | LXIN |
| 0006 I | 001156 | LXIV | 0006 | 1 | 001172 | LYIN | 0006 | 1 | 001206 | LYIV | 0006 | I | 026164 | M | 0006 | I 001222 | METER |
| 0007 I | 000031 | MS | 0004 | 1 | 000006 | N | 0005 | 1 | 000114 | NETA | 0006 | I | 001223 | NJOUL | 0005 | 000115 | NNLEO |
| 0006 I | 026165 | NOGRID | 0007 | I | 000032 | NPCON | 0007 | I | 000033 | NPLOT | 0005 | I | 000120 | NS | 0007 | I 000072 | NSTAT |
| 0007 I | 000154 | NSTP | 0000 | I | 000002 | NUMBR | 0006 | I | 001224 | OULES | 0006 | R | 020266 | 0 | 0006 | R 023222 | R |
| 0006 R | 026156 | RCIRC | 0006 | R | 026157 | RSQAR | 0006 | R | 026160 | RSTAR | 0006 | I | 001225 | S | 0006 | I 001226 | SECON |
| 0006 I | 001227 | SPER | 0006 | I | 001230 | SQUARE | 0006 | R | 001231 | U | 0006 | R | 004165 | $v$ | 0006 | R 007121 | W |
| 0006 I | 012055 | WATTS | 0006 | R | 012056 | $X$ | 0006 | $R$ | 012224 | XL | 0006 | R | 012222 | XMAX | 0006 | R 012223 | XMIN |
| 0006 R | 012225 | XR | 0006 | R | 012226 | Y | 0006 | R | 015244 | YB | 0006 | R | 015245 | YMAX | 0006 | 015246 | YMIN |
| $0006 R$ | 015247 | YT | 0006 | R | 015250 | $z$ | 0006 | I | 020204 |  |  |  |  |  |  |  |  |





@SYS\$*MSFCFOR\$.FOR, IS ROUGH
HSA E3-12/10/84-22:23:44 (.0)

```
SUBROUTINE ROUGH ENTRY POINT OOO172
    STORAGE USED: CODE (1) 000176; DATA(O) O00034; BLANK COMMON(2) 0000000
COMMON BLOCKS:
\begin{tabular}{lll}
0003 & RUF & 000022 \\
0004 & RUF3 & 000002
\end{tabular}
```

EXTERNAL REFERENCES (BLOCK, NAME)

| 0005 | ALOG 10 |
| ---: | ---: |
| 0006 | XPRR |
| 0007 | SQRT |
| 0010 | NERR3 |

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)



| 00105 | 25* | C | 1.15 - AXISYMETRIC | 000000 |
| :---: | :---: | :---: | :---: | :---: |
| 00105 | 26* | C |  | 000000 |
| 00105 | 27* | C |  | 000000 |
| 00105 | 28* | C | OUTPUT VARIABLES | 000000 |
| 00105 | 29* | C | - - | 000000 |
| 00105 | 30* | C | ST - ROUGH WALL STANTON NUMBER | 000000 |
| 00105 | 31* | C | PCT - PERCENT OF TRANSITION TO FULLY ROUGH | 000000 |
| 00105 | 32* | C | CF - ROUGH WALL SKIN FRICTION COEFFICIENT | 000000 |
| 00105 | 33* | C |  | 000000 |
| 00106 | 34* |  | IF (ICF.EQ.3)GO TO 100 | 000000 |
| 00106 | 35* | C | SKIN FRICTION COMPRESSIBILITY (YOUNG) | 000000 |
| 00110 | 36* |  | CFCFI $=(0.365 * H E / H A W)+(0.635 * H E / H W)$ | 000002 |
| 00111 | 37* |  | IF (CFCFI. LE. 0.0 ) CF CFI $=0.0$ | 000012 |
| 00111 | 38* | C | INCOMPRESSIBLE ROUGH WALL SKIN FRICTION | 000012 |
| 00113 | 39* |  | IF (ICF NE, 1)GO TO 10 | 000016 |
| 00113 | 40* | C | OPTION( 1 ) PRANDTL - SCHLICHTING | 000016 |
| 00115 | 41* |  | CFI $=(2.87+1.58 * A L O G 10(X / R K)) * *-2.5$ | 000021 |
| 00116 | 42* |  | GO TO 20 | 000037 |
| 00117 | 43* | 10 | CONTINUE | 000041 |
| 00117 | 44* | C | OPTION( 2) DROBLENKOV | 000041 |
| 00120 | 45* |  | CFI $=0.0139 *(\mathrm{X} / \mathrm{RK}) * *-(1.0 / 7.0)$ | 000041 |
| 00121 | 46* | 20 | CONTINUE | 000052 |
| 00122 | 47* |  | CFR $=$ CFCFI $* C F I * F M F$ | 0000052 |
| 00122 | 48* | C | TRANSITION CRITERION (FENTER) | 000052 |
| 00123 | 49* |  | UTAU $1=U E * S Q R T((C F R / 2 . O) *(R H O E / R H O W))$ | 000055 |
| 00124 | 50* |  | ETAK $=$ RHOW * UTAU1*RK/MUW | 000070 |
| 00124 | 51* | C | ROUGH SURFACE TURBULENT STANTON NUMBER | 000070 |
| 00124 | 52* | C | $A=0.52$ NOMINAL , RANGE OF 0.45 TO 0.7 (OWEN - THOMSON) | 000070 |
| 00125 | 53* |  | REK $=$ RHOE *UE*RK/MUE | 000074 |
| 00126 | 54* |  | STR=CFR/2.*(1.+A*(CFR/2.)**.725*REK**.45*PR**.8)**-1. © (SEIDMAN) | 000101 |
| 00126 | 55* | C | STR $=$ CFR/2.*(1.+A* ( $\mathrm{CFR} / 2) *.(H W / H E)$ )**. $5 * R E K * * .45 * P R * * .8) * *-1$. @ (HILL) | 000101 |
| 00126 | 56* | C |  | 000101 |
| 00126 | 57* | C | 100 USED BY FENTER, 70 USED BY HILL, 65 USED BY PIMENTA | 000101 |
| 00126 | 58* | C | PIMENTA VALUE CURRENTLY USED FOR TRANSITION | 000101 |
| 00126 | 59* | C | ETAK LE. 5.0 SMOOTH | 000101 |
| 00126 | 60* | C | $5.0 . L E . E T A K . L E . ~ 65.0 ~ T R A N S I T I O N A L L Y ~ R O U G H ~$ | 000101 |
| 00126 | 61* |  | 65.0.LT. ETAK ROUGH | 000101 |
| 00126 | 62* | C |  | 000101 |
| 00127 | 63* |  | PCT $=(E T A K-5.0) /(65.0-5.0)$ | 000126 |
| 00130 | 64* |  | IF (PCT.LT.O.O)PCT $=0.0$ | 000132 |
| 00132 | 65* |  | $I F(P C T . G T, 1.0) P C T=1.0$ | 000136 |
| 00134 | 66* |  | $C F=(P C T * C F R)+((1 . O-P C T) * C F S)$ | 000144 |
| 00135 | 67* |  | $S T=(P C T * S T R)+((1.0-P C T) * S T S)$ | 000154 |
| 00136 | 68* | 100 | CONTINUE | 000163 |
| 00137 | 69* |  | RETURN | 000163 |
| 00140 | 70* |  | END | 000175 |

```
@SYS$*MSFCFOR$.FOR,IS PARTCL
HSA E3 -12/10/84-22:23:46 (.0)
```

SUBROUTINE PARTCL ENTRY POINT 000165

STORAGE USED: CODE (1) OOO170; DATA(O) 000026; BLANK COMMON (2) 000000
COMMON BLOCKS:

| 0003 | PARTI | 000020 |
| :--- | :--- | :--- |
| 0004 | RUF | 000022 |

EXTERNAL REFERENCES (BLOCK, NAME)

| 0005 | SQRT |
| :---: | :---: |
| 0006 | XPRR |
| 0007 | NERR3 |

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)


| 00101 | 1* | SUBRDUTINE PARTCL |  |  | 000000 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 00103 | 2* | REAL K,KG, LAMBV,M,MUE, NU |  |  | 000000 |
| 00104 | 3* | COMMON /PARTI/M, LAMBV,RHOP,RED, CP, WP, CF, WF, KG, TAW, TW, |  |  | 000000 |
| 00104 | 4* | \$ ILT, Q, K, RP, IPART |  |  | 000000 |
| 00105 | 5* |  |  |  | 000000 |
| 00105 | 6* | 1 DUMM9, MUE, DUMM 11 , UE, RK, ICF, FMF, STR, DUMM 17 , CFR |  |  | 000000 |
| 00105 | 7* | C |  |  | 000000 |
| 00105 | 8* | C | INPUT VARIABLES |  | 000000 |
| 00105 | 9* | C |  |  | 000000 |
| 00105 | 10* | C | M - AVERAGE PARTICLE MASS | (LBM) | 000000 |
| 00105 | 11* | C | UE - BOUNDARY LAYER EDGE VELOCITY | (FT/SEC) | 000000 |
| 00105 | 12* | C | SIGMA - STOKERS DRAG COEFFICIENT | (LBM/SEC) | 000000 |
| 00105 | 13* | C | A - RADIUS OF SPHERICAL PARTICLE | (FT) | 000000 |
| 00105 | 14* | C | MUE - GAS VISCOSITY | (LBM/FT SEC) | 000000 |
| 00105 | 15* | C | LAMBV - MOMENTUM RANGE | (FT) | 0000000 |
| 00105 | 16* | C | RHOP - PARTICLE MASS DENSITY OF THE GAS | (LBM/FT3) | 000000 |
| 00105 | 17* | C | RHOE - GAS DENSITY | (LBM/FT3) | 000000 |
| 00105 | 18* | C | RED - EDGE REYNOLDS NUMBER BASED ON D |  | 000000 |
| 00105 | 19* | C | X - RUNNING LENGTH | (FT) | 000000 |
| 00105 | 20* | C | TAU - SHEAR STRESS | (LBF/FT2) | 000000 |
| 00105 | 21* | C | CFRO - FRICTION COEFFICIENT |  | 000000 |
| 00105 | 22* | C | STS - SMOOTH STANTON NUMBER |  | 000000 |


| 00105 | 23* | C CP - SPECIFIC HEAT OF THE SOLID PARTICLE (BTU/LB DEG F) | 000000 |
| :---: | :---: | :---: | :---: |
| 00105 | 24* | C WP - MASS FLOW OF PARTICLES (LB/FT2 SEC) | 000000 |
| 00105 | 25* | C CF - SPECIFIC HEAT AT CONSTANT PRESSURE OF FLUID (BTU/LB DEG F) | 000000 |
| 00105 | 26* | C WF - MASS FLOW OF FLUID. $\quad$ (LB/FT2 SEC) | 000000 |
| 00105 | 27* | C K-RATIO OF PARTICLE DENSITY TO FLUID MASS DENSITY AT EDGE | 000000 |
| 00105 | 28* | C KG - THERMAL CONDUCTIVITY OF THE GAS (BTU/SEC FT DEG K) | 000000 |
| 00105 | 29* | C D - DIAMETER OF THE TUBE | 000000 |
| 00105 | 30* | C NU - NUSSELT'S NUMBER | 000000 |
| 00105 | 31* | C TAW - ADIABATIC WALL TEMPERATURE $\quad$ DEG. R | 000000 |
| 00105 | 32* | C TW - WALL TEMPERATURE $\quad$ DEG. R | 000000 |
| 00105 | 33* | C ILT - FLOW TYPE FLAG 1 - LAMINAR | 000000 |
| 00105 | $34 *$ | C 2-TURBULENT | 000000 |
| 00105 | 35* | C OUTPUT VARIABLES | 0000000 |
| 00105 | 36* | C | 000000 |
| 00105 | 37* | C CFR - MODIFIED FRICTION COEFFICIENT | 000000 |
| 00105 | 38* | C STR - PARTICLE STANTON NUMBER | 000000 |
| 00105 | 39* | C | 000000 |
| 00106 | 40* | $\mathrm{PI}=3.1415927$ | 000000 |
| 00107 | 41* | IF (ILT.EQ.2) GOTO 100 | 000001 |
| 00107 | 42* | C IF ILT $=2$ THE FLOW IS TURBULENT | 000001 |
| 00111 | 43* | K=RHOP / RHOE | 000004 |
| 00112 | 44* | Y $=$ LAMBV/X | 000007 |
| 00112 | 45* | C THE EQUATIONS USED TO COMPUTE QDOT AND CFR ARE DIFFERENT WHEN | 000007 |
| 00112 | 46* | C LAMBDA /X IS LESS THAN 1. THAT THE EQUATIONS USED WHEN LAMBDA/X | 000007 |
| 00112 | 47* | C IS GREATER THAN 1. HERE ONLY LAMBDA/X LESS THAN 1 CASE IS USED. | 0000007 |
| 00113 | 48* | $10 \quad C F R=C F R O * S Q R T(1 .+K) *(1 .+(.49 *(Y * K /(1 .+K)))$ ) | 000012 |
| 00114 | 49* | STR $=$ STS*SQRT $(1 .+K) *(1 .+(.49 *(Y * K /(1 .+K)))$ | 000031 |
| 00115 | 50* | $Q=S T R / S T S$ | 000035 |
| 00116 | 51* | GOTO 160 | 000037 |
| 00117 | $52 *$ | 100 BETAS $=(W P * C P) /(W F * C F)$ | 000041 |
| 00120 | 53* | $W=W P / W F$ | 0000046 |
| 00121 | 54* | IF (W.LT. 1. OR.ABS (W-1.).LT..001) GO TO 105 | 000051 |
| 00123 | 55* | IF (W.GT. 1.) GOTO 110 | 000070 |
| 00123 | 56* | C THIS IF STATEMENT SERVES THE SAME PURPOSE AS THE IF STATEMENT FOR | 000070 |
| 00123 | 57* | C THE LAMINAR CASE | 000070 |
| 00125 | $58 *$ | 105 CFR $=$ CFRO* (1.+BETA5) | 000075 |
| 00126 | 59* | STR $=$ STS * ( $1 .+$ BETA5) | 000101 |
| 00127 | 60* | $0=$ STR/STS | 000104 |
| 00130 | 61* | GOTO 120 | 000106 |
| 00131 | 62* | 110 NU=. $14 *($ RED **.6)*(W**.45) | 000110 |
| 00132 | 63* | $\mathrm{D}=$ RED / (RHOE * UE/MUE) | 000123 |
| 00133 | 64* | QDOT $=((N U * K G) / D) *(T A W-T W)$ | 000131 |
| 00134 | 65* | STR = QDOT/( (DUMM5-DUMM6)*RHOE*UE) | 000137 |
| 00135 | $66 *$ | $Q=S T R / S T S$ | 000145 |
| 00136 | 67* | CFR $=$ Q*CFRO | 000147 |
| 00137 | $68 *$ | 120 CONT INUE | 000152 |
| 00140 | 69* | RETURN | 000152 |
| 00141 | 70* | 160 CONT INUE | 000155 |
| 00142 | 71* | RETURN | 000155 |
| 00143 | 72* | END | 000167 |

## @SYS\$*MSFCFOR\$.FOR,WUS BLKDTA.

HSA E3 -12/10/84-22:23:49 (22,23)
$-21$
COMMON/RUF DUMM1, DUMM2, DUMM3,DUMM4 DUMM5, DUMM6, DUMM7, DUMM8, DUMM9
\$ DUMM 10, DUMM 11 , DUMM 12, RK, ICF, FMF, DUMM16, DUMM 17 , DUMM 18
COMMON/PART I/PARTM, DUMM24, RHOPA, DUMM23, CPART, WP, DUMM19, WF.
\$ DUMM22, DUMM2O, DUMM21, ILT, PF, AK, RP, IPART
COMMON /LAM/ LLAMIN
$-92$
C DEFAULT VALUES FOR ROUGHNESS OPTION
DATA FMF/1.15/.ICF/O/.RK/O.O/
C DEFAULT VALUES FOR PARTICLE OPTION
DATA IPART/O/.RP/O./.WP/O./.WF/1./.CPART/O./.RHOPA/O./
OEFAULT VALUES FOR RELAMINARIZATION OPTION
DATA ILAMIN/O/
BLOCK DATA
STORAGE USED: CODE (1) OOOOOO; DATA (O) OOOOOO; BLANK COMMON(2) OOOOOO
COMMON BLOCKS :

| 0003 | AL | 000002 |
| :---: | :---: | :---: |
| 0004 | CARDS | 000003 |
| 0005 | CONSTS | 000010 |
| 0006 | CRBCOM | 000111 |
| 0007 | EPSCOM | 000045 |
| 0010 | EQPCOM | 000435 |
| 0011 | ETACOM | 000017 |
| 0012 | HOLLER | 000060 |
| 0013 | INPUTI | 000015 |
| 0014 | INTCOM | 000115 |
| 0015 | LOWTH | 001372 |
| 0016 | NZERO | 000001 |
| 0017 | PLOTS | 000172 |
| 0020 | PRMALS | 000154 |
| 0021 | RFTCOM | 000045 |
| 0022 | RUF | 000022 |
| 0023 | PARTI | 000020 |
| 0024 | LAM | 000001 |
| 0025 | SAHA | 000066 |
| 0026 | TEMCOM | 000162 |
| 0027 | UNICOM | 000011 |
| 0030 | WALTEM | 000715 |

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

| 0020 |  | 000000 | A | 0023 |  | 000015 | AK | 0030 |  | 000000 | ALTAB | 0012 | I | 000000 | AREA | 0006 |  | 000000 | ASU |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0010 |  | 000000 | ATA | 0010 | R | 000030 | BASMOL | 0014 |  | 000000 | CASE | 0014 | R | 000015 | CBAR | 0007 | R | 000000 | CLNUM |
| 0023 | R | 000004 | CPART | 0005 | R | 000000 | CPFL | 0025 |  | 000000 | CPH | 0015 | R | 000000 | CPL | 0012 | I | 000002 | CO |
| 0012 | I | 000006 | DENS | 0012 | I | 000010 | DIST | 0007 |  | 000001 | DL | 0005 | R | 000001 | DPR | 0004 |  | 000000 | DUB8 |
| 0022 |  | 000000 | DUMM 1 | 0022 |  | 000011 | DUMM10 | 0022 |  | 000012 | DUMM1 1 | 0022 |  | 000013 | DUMM 12 | 0022 |  | 000017 | DUMM16 |
| 0022 |  | 000020 | DUMM17 | 0022 |  | 000021 | DUMM18 | 0023 |  | 000006 | DÜM 19 | 0022 |  | 000001 | DUMM2 | 0023 |  | 000011 | DUMM2O |
| 0023 |  | 000012 | DUMM2 1 | 0023 |  | 000010 | DUMM22 | 0023 |  | 000003 | DUMM23 | 0023 |  | 000001 | DUMM24 | 0022 |  | 000002 | DUMM3 |
| 0022 |  | 000003 | DUMM4 | 0022 |  | 000004 | DUMM5 | 0022 |  | 000005 | DÜMG | 0022 |  | 000006 | DUMM 7 | 0022 |  | 000007 | DUMM8 |
| 0022 |  | 000010 | DUMM9 | 0007 | R | 000020 | ELCON | 0012 | I | 000012 | ENERGY | 0010 | R | 000031 | EPOVRK | 0007 |  | 000021 | EPSA |
| 0011 | R | 000000 | ETA | 0010 |  | 000032 | FF | 0004 | R | 000001 | FFAR | 0004 | R | 000002 | FITMOL | 0017 |  | 000000 | FLDX |
| 0012 | I | 000014 | FLUX | 0022 | R | 000016 | FMF | 0021 | R | 000000 | F2FIX | 0021 |  | 000017 | F2FIXT | 0005 | $R$ | 000002 | GC |
| 0012 | I | 000016 | HEAT | 0015 | R | 000226 | HL | 0012 | I | 000022 | HWALL | 0014 |  | 000016 | I | 0015 | I | 000454 | IADD |


| 0013 | I 000000 | IBODY | 0022 | I | 000015 | ICF | 0025 | 1 | 000062 | I CON | 0030 | I | 000231 | ICOOL | 0026 | I | 000000 | IDAT |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0013 | I 000001 | IDERIV | 0013 | I | 000002 | IDIFF | 0026 | I | 000001 | IED | 0030 |  | 000232 | IENH | 0013 | 1 | 000003 | IETA |
| 0013 | 1000004 | IFIT | 0013 | I | 000005 | IFLOW | 0013 | I | 000006 | IGUESS | 0024 | I | 000000 | ILAMIN | 0015 | I | 000455 | ILSP |
| 0023 | 000013 | ILT | 0026 | I | 000002 | IND | 0015 | I | 000537 | INEW | 0026 | I | 000003 | IOR | 0026 | I | 000004 | IOUT |
| 0023 | I 000017 | IPART | 0025 | I | 000063 | IPASS | 0003 | 1 | 000000 | IPLOT | 0017 | I | 000030 | IPP | 0020 | 1 | 000146 | IPUNCH |
| 0026 | I 000005 | IRE | 0025 | I | 000064 | IRITE | 0026 | 1 | 000006 | IROC | 0026 | I | 000007 | ISAV | 0026 | I | 000010 | ITA |
| 0013 | 1 1 000007 | ITDK | 0026 | 1 | 000011 | ITE | 0013 | I | 000010 | ITHERM | 0025 | I | 000065 | ITRCNT | 0003 | I | 000001 | IUNIT |
| 0013 | 1000011 | IWALL | 0020 |  | 000147 | J | 0013 | I | 000012 | JWALL | 0014 | I | 000023 | KAPPA | 0013 | I | 000013 | KEDGP |
| 0014 | I 000024 | KONRFT | 0014 |  | 000025 | KR9 | 0026 | I | 000012 | LIF | 0026 | I | 000014 | LLA | 0026 | I | 000015 | LLAW |
| 0026 | I 000016 | LOT | 0026 | I | 000017 | MAIN | 0012 | I | 000024 | MASS | 0026 | I | 000020 | MOEG | 0017 | I | 000031 | MS |
| 0026 | I 000021 | MURD | 0026 | I | 000022 | NCH | 0014 | 1 | 000114 | NETA | 0026 | 1 | 000023 | NIT | 0015 | I | 000551 | NLTSP |
| 0017 | I 000032 | NPCON | 0017 | 1 | 000033 | NPLOT | 0021 | I | 000043 | NPOINT | 0017 | I | 000071 | NSTAT | 0017 | I | 000153 | NSTP |
| 0026 | I 000024 | NTOR | 0013 | I | 000014 | NTROPY | 0016 | 1 | 000000 | NUL | 0026 | I | 000026 | NW | 0023 |  | 000000 | PARTM |
| 0005 | R 000003 | PATM | 0023 |  | 000014 | PF | 0005 | R | 000004 | PI | 0020 | R | 000153 | PNORM | 0012 | I | 000030 | PRESS |
| 0007 | R 000040 | PRT | 0012 | I | 000032 | RAD | 0021 | R | 000044 | RATLIM | 0005 | R | 000005 | RBAR | 0007 | R | 000041 | RETR |
| 0012 | 1000034 | REY | 0023 | R | 000002 | RHOPA | 0007. |  | 000042 | RHOVS | 0022 | R | 000014 | RK | 0023 | R | 000016 | RP |
| 0005 | R 0000006 | RVAR | 0007 | R | 000043 | SCT | 0012 | I | 000036 | SHEAR | 0010 | R | 000434 | SIGMA | 0005 | R | 000007 | SIPSF |
| 0015 | R 000552 | SL | 0006 | R | 000110 | STEF | 0015 | I | 001000 | SUBLT | 0012 | I | 000040 | TCON | 0012 | I | 000044 | TEMP |
| 0012 | I 000046 | THRUST | 0015 | R | 001144 | TL | 0030 | R | 000714 | TOLOW | 0027 | R | 000000 | UCD | 0027 | R | 000001 | UCE |
| 0027 | R 000002 | UCL | 0027 | R | 000003 | UCM | 0027 | R | 000004 | UCP | 0027 | R | 000005 | UCR | 0027 | R | 000006 | UCS |
| 0027 | R 0000007 | UCT | 0027 | R | 000010 | UCV | 0012 | I | 000050 | VEL | 0012 | I | 000052 | VIS | 0023 | R | 000007 | WF |
| 0023 | R 000005 | WP | 0012 | I | 000056 | XIPR | 0007 | R | 000044 | YAP |  |  |  |  |  |  |  |  |





END OF COMPILATION: NO DIAGNOSTICS.
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## EXTERNAL REFERENCES (BLOCK, NAME)



| 0015 | 1 | 000004 | IFIT | 0015 | 1 | 000005 | IFLOW | 0005 | 1 | 000003 | IFRAC | 0015 | I | 000006 | IGUESS | 0026 |  | 000001 | ILAM |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0032 | 1 | 000000 | ILAMIN | 0020 | I | 000455 | ILSP | 0031 |  | 000013 | ILT | 0046 | I | 000002 | IND | 0020 | I | 000537 | INEW |
| 0035 |  | 000000 | INT | 0046 | I | 000003 | IOR | 0046 | I | 000004 | IOUT | 0023 | I | 000145 | IP | 0031 | I | 000017 | IPART |
| 0034 | I | 000063 | IPASS | 0003 | I | 000000 | IPLOT | 0022 | I | 000030 | IPP | 0023 | I | 000146 | IPUNCH | 0016 | I | 000017 | IQ |
| 0012 |  | 000316 | IR | 0046 | I | 000005 | IRE | 0034 | I | 000064 | IRITE | 0046 | I | 000006 | IROC | 0016 | I | 000020 | IS |
| 0046 | I | 000007 | ISAV | 0016 | I | 000021 | ISH | 0013 | I | 000000 | ISN | 0023 | I | 000147 | IST | 0007 |  | 000025 | ISU |
| 0046 | I | 000010 | ITA | 0051 | 1 | 000233 | ITCOOL | 0015 | I | 000007 | ITDK | 0046 | 1 | 000011 | ITE | 0005 | I | 000004 | ITEMP |
| 0036 |  | 000002 | ITFF | 0015 | I | 000010 | ITHERM | 0051 | I | 000234 | ITHICK | 0051 | I | 000235 | ITLINP | 0034 |  | 000065 | ITRCNT |
| 0016 |  | 000022 | ITS | 0023 | I | 000150 | IU | 0003 | I | 000001 | IUNIT | 0015 | I | 000011 | IWALL | 0000 | I | 000027 | $J$ |
| 0040 |  | 000052 | UJ | 0015 | I | 000012 | JWALL | 0016 | 1 | 000023 | KAPPA | 0025 |  | 000036 | KAPPAL | 0025 | I | 000037 | KAPPAT |
| 0012 | I | 000326 | KAT | 0017 |  | 0000000 | KBC | 0017 |  | 000001 | KCC | 0015 | I | 000013 | KEDGP | 0016 | I | 000024 | KONRFT |
| 0017 | I | 000003 | KQ10 | 0017 |  | 000002 | K09 | 0036 |  | 000003 | KR2 | 0016 | I | 000025 | KR9 | 0007 | I | 000026 | KS |
| 0017 |  | 000004 | KSB | 0017 |  | 000005 | KSOL | 0025 | I | 000040 | KTURB | 0017 |  | 000006 | KT8 | 0005 | I | 000005 | KU |
| 0051 | R | 000236 | LAMDAW | 0012 |  | 000336 | LAMI | 0051 | R | 000237 | LAMTAB | 0014 |  | 000126 | LAR | 0004 | I | 001606 | LEF |
| 0004 |  | 001616 | LEFS | 0004 |  | 001626 | LEFT | 0004 |  | 001636 | LEFW | 0046 | I | 000012 | LIF | 0046 | I | 000014 | LLA |
| 0046 | I | 000015 | LLAW | 0046 | I | 000016 | LOT | 0000 | L | 000016 | LOWT | 0004 |  | 001604 | L2 | 0004 |  | 001605 | L3 |
| 0046 | I | 000017 | MAIN | 0016 |  | 000107 | MAT 11 | 0016 |  | 000110 | MAT 1 J | 0016 |  | 000111 | MAT2I | 0004 |  | 001646 | MOA |
| 0004 |  | 001742 | MOB | 0046 | I | 000020 | MOEG | 0000 | R | 000017 | MR | 0022 | I | 000031 | MS | 0046 | I | 000021 | MURD |
| 0016 | 1 | 000112 | MWE | 0005 | I | 000006 | N | 0016 |  | 000113 | NAM | 0045 | I | 000026 | NC | 0000 | I | 000020 | NCASE |
| 0046 | I | 000022 | NCH | 0036 |  | 000004 | NCV | 0012 | I | 000432 | NEL | 0016 | I | 000114 | NETA | 0025 |  | 000041 | NETAL |
| 0025 | I | 000042 | NETAT | 0005 | I | 000007 | NFF | 0046 | I | 000023 | NIT | 0020 |  | 000551 | NLTSP | 0016 |  | 000115 | NNLEQ |
| 0016 | I | 000116 | NON | 0005 | I | 000010 | NP | 0022 | I | 000032 | NPCON | 0022 | I | 000033 | NPLOT | 0025 | I | 000043 | NPOINT |
| 0016 |  | 000117 | NRNL | 0016 | 1 | 000120 | NS | 0034 | I | 000066 | NSU | 0016 | I | 000121 | NSP | 0004 |  | 002036 | NSPEC |
| 0016 |  | 000122 | NSPM1 | 0022 | I | 000071 | NSTAT | 0022 | I | 000153 | NSTP | 0000 | I | 000021 | NTAL | 0023 | I | 000151 | NTH |
| 0046 | I | 000024 | NTOR | 0015 | 1 | 000014 | NTROPY | 0046 | I | 000026 | NW | 0000 | R | 000025 | OK | 0000 |  | 001106 | OUT |
| 0000 | R | 000007 | OX | 0012 |  | 000433 | P | 0031 |  | 000000 | PARTM | 0023 | R | 000152 | PCHAMB | 0010 |  | 000065 | PE |
| 0031 |  | 000014 | PF | 0004 |  | 002037 | PIEASE | 0043 |  | 000044 | PIM | 0021 | R | 013560 | PITAB | 0043 |  | 000045 | PM |
| 0023 | R | 000153 | PNORM | 0045 | R | 000027 | PRA | 0045 | R | 000030 | PRB | 0045 | R | 000031 | PRC | 0045 | R | 000032 | PRD |
| 0045 | R | 000033 | PRDUM | 0024 | R | 000063 | PRE | 0011 | $R$ | 000040 | PRT | 0013 |  | 000074 | PVMW | 0013 |  | 000120 | PVOL |
| 0034 |  | 000067 | QWG | 0023 | R | 000156 | RADFL | 0024 | R | 000145 | RADR | 0024 |  | 000227 | RADS | 0023 |  | 000154 | RAD5 |
| 0023 |  | 000155 | RAD6 | 0025 | R | 000044 | RATLIM | 0051 | R | 000321 | RECOFT | 0043 |  | 000046 | RED | 0027 |  | 0000000 | RE THMO |
| 0011 | R | 000041 | RETR | 0010 |  | 000147 | RHOE | 0031 | $R$ | 000002 | RHOPA | 0011 |  | 000042 | RHOVS | 0050 | R | 000372 | RHOVW |
| 0030 | R | 000014 | RK | 0024 | R | 000311 | ROKAP | 0031 | R | 000016 | RP | 0023 | R | 000157 | RTM | 0024 | R | 000373 | S |
| 0011 | R | 000043 | SCT | 0035 |  | 000001 | SDRHOH | 0035 |  | 000002 | SDRHOK | 0012 | R | 000434 | SIGMA | 0020 | R | 000552 | SL |
| 0047 | R | 000152 | SP | 0026 |  | 000002 | SPCT | 0010 |  | 000231 | SPE | 0005 |  | 000072 | SPL | 0005 |  | 000073 | SPU |
| 0050 | R | 000454 | SPW | 0007 |  | 000110 | STEEF | 0020 | I | 001000 | SUBLT | 0023 |  | 000160 | SUMQG | 0012 |  | 000435 | T |
| 0010 |  | 000705 | TE | 0012 | R | 001006 | TF | 0051 | R | 000322 | THITAB | 0051 |  | 000404 | TI | 0013 | R | 000144 |  |
| 0005 | R | 000074 | TJA | 0012 | R | 000436 | TKP | 0013 | R | 000147 | TKT | 0020 | R | 001144 | TL | 0051 | R | 000466 | TLINP |
| 0051 | R | 000632 | TLTAB | 0051 | R | 000714 | TOLQW | 0012 |  | 000526 | TQ | 0045 |  | 000034 | TR | 0043 |  | 000047 | TREF |
| 0010 |  | 000767 | TTVC | 0012 |  | 000616 | TU | 0051 | R | 000715 | TUBEN | 0010 |  | 000770 | TVCC | 0050 | R | 001130 |  |
| 0010 |  | 001052 | UE | 0042 |  | 000001 | UKAPPA | 0033 |  | 000000 | UTAU | 0021 |  | 016514 | VA | 0043 |  | 000050 | VINTR |
| 0021 | R | 021450 | VITAB | 0045 | R | 000037 | VMUA | 0045 | R | 000040 | VMUB | 0045 | R | 000041 | VMUC | 0045 | R | 000042 | VMUD |
| 0010 |  | 001134 | VMUE | 0045 |  | 000043 | VMWD | 0010 |  | 001216 | VMWE | 0012 |  | 001102 | VN | 0012 |  | 001176 | VNU |
| 0021 |  | 024404 | VS | 0004 |  | 002040 | W | 0042 |  | 000020 | WALLA | 0012 | R | 002136 | WAT | 0031 | R | 000007 | WF |
| 0012 |  | 002146 | WM | 0013 | R | 000217 | WMS | 0031 | R | 000005 | WP | 0012 |  | 002147 | WTM | 0051 | R | 000716 | XAREA |
| 0051 | R | 001000 | XENH | 0021 | R | 027340 | XITAB | 0051 | R | 001062 | XLTAB | 0023 |  | 000161 | XST | 0051 | R | 001226 | XTHIK |
| 0012 |  | 002243 | Y | 0011 | R | 000044 | YAP | 0021 | R | 032274 | YITAB | 0012 |  | 002337 | YW | 0051 | R | 001310 | ZMUTAB |
| 0000 | R | 000022 | ZP |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |


| 00100 | 1* | CBLIMP | BOUNDARY LAYER INTEGRAL MATRIX PROCEDURE | BLIM 001 | 000000 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 00100 | $2 *$ | C |  |  | 000000 |
| 00101 | 3* | COMMON | /AL/ IPLOT. IUNIT | /AL/ | 000000 |
| 00103 | 4* | COMMON | /BLQCOM/ FR(60, 15),L2,L3.LEF (8), LEFS(8), LEFT (8), LEFW (8). | /BLQCOM/ | 000001 |
| 00103 | 5* | 1 | MOA(60), MOB(60),NSPEC,PIEASE,W(3) | /BLOCOM/ | 000001 |
| 00104 | 6* | Соммо̃ | /CARDS/ DUB8, FFAR, FITMOL, IFRAC, ITEMP, KU, N, NFF, NP (50), SPL, | /CARDS/ | 000001 |
| 00104 | 7* | 1. | SPU, TUA (3) | /CARDS/ | 000001 |
| 00105 | $8 *$ | COMMON | /COEFFS/ COEF(7,3,60) | /COEFFS/ | 000001 |




| 00171 | 129* |  | 5 IDAT,NIT,MURD, IOR, NCH, ISAV, ITA, IROC, ITE, LLA , NCH, IDAT | ANK 7/83 | 000001 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 00307 | 130* | 46 | MWE $=-1$ |  | 000137 |
| 00310 | 131* |  | READ (5,1,END=50) CASE |  | 000140 |
| 00313 | 132* | 1 | FORMAT (13A6) |  | 000151 |
| 00314 | 133* |  | READ (5, DATA , END $=50$ ) |  | 000151 |
| 00317 | 134* |  | IF (ZP.GT. 0.0) WRITE (6, ARRAYS) | ANK 3/83 | 000156 |
| 00323 | 135* |  | IF (ZP, GE - O.O) WRITE (6,OUT) | ANK 3/83 | 000165 |
| 00327 | 136* |  | IF (NEL .LE. 8 . AND. NSP .LE. 7) GO TO 25 | ANK $8 / 83$ | 000174 |
| 00331 | 137* |  | WRITE (6,3) NEL,NSP | ANK 8/83 | 000212 |
| 00335 | 138* | 3 | FORMAT (//10X, 'NEL MUST BE .LE. 8 , NEL $=0,12$, | ANK 8/83 | 000221 |
| 00335 | 139* |  | 1, NSP MUST BE.LE. 7. NSP $=$ ', I2//) | ANK 8/83 | 000221 |
| 00336 | 140* |  | NEL $=$ MINO(NEL, 8) | ANK $8 / 83$ | 000221 |
| 00337 | 141* |  | NSP $=$ MINO(NSP, 7) | ANK 8/83 | 000227 |
| 00340 | 142* | 25 | IF (MR LE. O.O) GO TO 30 | ANK $8 / 83$ | 000236 |
| 00342 | 143* |  | IF (NTAL.GT. 0 ) $\mathrm{GE}=(\mathrm{MR} * \mathrm{HOX}+\mathrm{HFU}) /(\mathrm{MR}+1.0)$ | ANK 4/83 | 000240 |
| 00342 | 144* | C | CALCULATE RELATIVE NUMBER OF ATOMS FROM THE MIXTURE RATIO AND THE |  | 000240 |
| 00342 | 145* | C | NUMBER OF EACH ELEMENT IN THE OXIDIZER AND FUEL |  | 000240 |
| 00344 | 146* |  | OK $=0.0$ | ANK 5/83 | 000255 |
| 00345 | 147* |  | $F K=0.0$ | ANK 5/83 | 000256 |
| 00346 | 148* |  | DO $10 \mathrm{~J}=1$, NSP | ANK $5 / 83$ | 000262 |
| 00351 | 149* |  | $F K=F K+F U E L(U) * W A T(U)$ | ANK 5/83 | 000262 |
| 00352 | 150* | 10 | OK $=$ OK + OX $(J) * W A T(J)$ | ANK $5 / 83$ | 000265 |
| 00354 | 151* |  | AMNO $=$ MR*FK/OK | ANK 5/83 | 000272 |
| 00355 | 152* |  | AMWP $=$ FK + AMNO*OK | ANK $5 / 83$ | 000275 |
| 00356 | 153* |  | AMNOS $=$ AMNO/AMWP | ANK 5/83 | 000300 |
| 00357 | 154* |  | AMNFS $=1.0 /$ AMWP | ANK 5/83 | 000302 |
| 00360 | 155* |  | DO $20 \mathrm{~J}=1$. NSP | ANK 5/83 | 000310 |
| 00363 | 156* | 20 | TKP $(u, 1)=A M N O S * O X(U)+A M N F S * F U E L(U)$ | ANK $5 / 83$ | 000310 |
| 00365 | 157* |  | CMR = AMNOS*OK/( AMNFS*FK) | ANK 5/83 | 000316 |
| 00366 | 158* |  | WRITE ( 6.2 ) CMR,MR | ANK 5/83 | 000324 |
| 00372 | 159* | 2 | FORMAT (//10X, 'COMPUTED MIXTURE RATIO $=$ ', F10.6,' INPUT MIXTURE' | ANK 5/83 | 000334 |
| 00372 | 160* |  | $1 \quad$ RATID $=, \quad \mathrm{F} 10.6 / 7)$ | ANK 5/83 | 000334 |
| 00373 | 161* | 30 | NCASE $=$ NCASE +1 | ANK 5/83 | 000334 |
| 00374 | 162* |  | NSJ $=15+$ NSP |  | 000336 |
| 00374 | 163* | C | IF LOW TEMPERATURE EXTENSION DATA HAS BEEN READ IN, CONVERT IT |  | 000336 |
| 00374 | 164* | C | TO INTERNALLY REQUIRED UNITS |  | 000336 |
| 00375 | 165* |  | IF (LOWT) CALL LTCPHS |  | 000341 |
| 00377 | 166* |  | IF (NCASE .GT. 1) REWIND 2 | ANK 7/83 | 000345 |
| 00401 | 167* |  | IF (IPLOT EQ. O) GO TO 45 |  | 000354 |
| 00403 | 168* |  | IF (NCASE .LE. 1) GO TO 35 | ANK $7 / 83$ | 000356 |
| 00405 | 169* |  | REWIND 3 |  | 000362 |
| 00406 | 170* |  | REWIND 4 |  | 000365 |
| 00407 | 171* | 35 | NSTAT(NS) $=1$ | ANK 7/83 | 000371 |
| 00410 | 172* |  | IF (IWALL . NE. 7) GO TO 45 | ANK $4 / 83$ | 000373 |
| 00412 | 173* |  | NPLOT(7) $=0$ |  | 000376 |
| 00413 | 174* |  | NPLOT(12) $=0$ |  | 000377 |
| 00414 | 175* | 45 | $I S=1$ | ANK 5/83 | 000401 |
| 00415 | 176* |  | $I Q=1$ |  | 000402 |
| 00416 | 177* |  | IF (ICOOL NE O . AND. ICON, EQ - O) IRITE = O |  | 000404 |
| 00420 | 178* |  | IF (IPLOT .EQ. O .OR. ICOOL .NE. O) IPASS $=1$ |  | 000415 |
| 00422 | 179* |  | IF (ICON EQ . 1 AND. IPLOT NE O O ) IPASS $=0$ |  | 000427 |
| 00424 | 180* | 41 | CALL SETUP |  | 000443 |
| 00425 | 181* | 43 | CALL ITERAT | BLIM 031 | 000445 |
| 00426 | 182* |  | CALL OUTPUT | BLIM 032 | 000446 |
| 00427 | 183* |  | IF (NON) 43,44, 5 |  | 000450 |
| 00432 | 184* | 44 | ISH $=$ IS | ANK $5 / 83$ | 000454 |
| 00433 | 185* |  | $10=10+1$ |  | 000456 |
| 00434 | 186* |  | $I S=I S+1$ | ANK $5 / 83$ | 000461 |
| 00435 | 187* |  | IF (NP(IS) EQ. NTH) NSTAT (IS) $=1$ | ANK 5/83 | 000464 |
| 00437 | 188* |  | IF (K010 + IS . EQ. -10) KQ10 $=1$ | ANK $5 / 83$ | 000474 |


| 00441 | 189* |  | IF (IS .EQ. NS ) IRITE $=1$ | ANK 5/83 | 000502 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 00443 | 190* |  | IF (IS .LE. NS) GO TO 41 | ANK 5/83 | 000507 |
| 00445 | 191* |  | $I S=N S$ | ANK 5/83 | 000513 |
| 00446 | 192* |  | IF (ICOOL.EQ. O.OR. ICON.EQ. 1) GO TO 15 |  | 000515 |
| 00450 | 193* |  | CALL SATEMP |  | 000527 |
| 00451 | 194* |  | GO TO 45 |  | 000531 |
| 00452 | 195* | 15 | IF (NP(IS) .LE NTH) GO TO 46 | ANK 8/83 | 000533 |
| 00454 | 196* |  | CALL ROCOUT | ANK 8/83 | 000537 |
| 00455 | 197* |  | IF (IPLOT GT. O) CALL PLOT |  | 000541 |
| 00457 | 198* |  | GO T0 46 |  | 000546 |
| 00460 | 199* | 50 | IF (IPUNCH NE - 1 . AND. IPUNCH NE. 2) CALL EXIT | ANK 8/83 | 000550 |
| 00462 | 200* |  | $J=0$ | ANK $8 / 83$ | 000565 |
| 00463 | 201* |  | WRITE (15) J | ANK 8/83 | 000566 |
| 00466 | 202* |  | END | BLIM 038 | 000575 |
| END OF COMPILATION: |  |  |  |  |  |

            COMMON/ACCN/ACCPK. ILAM, SPCT
            COMMON/ACPK/ACCPK 1, ACCPK2
            COMMON/RETH/RETHMO
            COMMON/RUF /DUMM 1 , DUMM2, DUMM3, DUMM4, DUMM5, DUMM6, DUMM 7, DUMM8, DUMM9,
            \$ DUMM 10, DUMM 11 , DUMM 12, RK, ICF,FMF, DUMM 16 , DUMM 17 , DUMM 18
            COMMON/PART I PARTM, DUMM24, RHOPA, DUMM23, CPART, WP, DUMM19,WF ,
            DUMM22, DÜMM2O, DUMM21, ILT, PF, AK, RP, IPART
            COMMON/LAM/ ILAMIN
    $-313$
RETHMO $=-$ C $3 M(I S) * \operatorname{RHOE}(I S) * U E(I S) * C T E * V M U E(I S) / V M U(N E T A)$
ACCP $=$ BETAV (IS) *VMUE (IS) **2*ROKAP (IS) $* * 2 / 2.0 / X I$ (IS)
ACCPK $=A C C P * R H O E(I S) * V M U(1) /(V M U E(I S) * R H O(1))$
IF (ILAMIN.EQ.O)GO TO 79
ILAM=0
IF(S(IS).GT.2.*STURB.AND.ACCPK.GT.1.1E-06)GO TO 69
GO TO 79
69 TF(RETHMO LT. 250.) GO T0 79
ILAM $=1$
$A A=8.935 E-14$
$B B=2.239 E-10$
$C C=1.0247 \mathrm{E}-06$
ACCPK $1=A A * R E T H M O * * 2+B B * R E T H M O+C C$
IF (RETHMO.LT.4100.) GO TO 98
I LAM $=0$
GO TO 99
98 GCCPK2=3.5E-06
99 IF $A C C P K . L T . A C C P K 1) I L A M=0$
IF $A C C P K \cdot L T \cdot A C C P K 1)$ I LAM $=0$
IF $A C C P K \cdot G T \cdot A C C P K 2)$ I LAM $=1$
CONTINUE
$-320$
-321
IF (RETHMO.GT.RETR)ILT=2
IF (RETHMO.ET.RETR $) I L T=1$
SUBROUTINE NNNCER ENTRY POINT 002554
NONCER ENTRY POINT 002557
STORAGE USED: CODE (1) 002562; DATA (0) 000142: BLANK COMMON(2) 000000
COMMON BLOCKS:
0003 BLQCOM 002043
0004 BUMCOM 000004
0005 COECOM 000014
0006 COECOM 000014
0007 CONSTS 000003
0010 CRBCOM 000110
0011 EDGCOM 001216
0012 EPSCOM 000042

| 0013 | EQPCOM | 000433 |
| :---: | :---: | :---: |
| 0014 | EQTCOM | 001427 |
| 0015 | ERRCOM | 000622 |
| 0016 | ETACOM | 002631 |
| 0017 | FLXCOM | 000020 |
| 0020 | HISCOM | 000426 |
| 0021 | INPUTI | 000015 |
| 0022 | INTCOM | 000123 |
| 0023 | INTERI | 000006 |
| 0024 | NONCOM | 035532 |
| 0025 | NZERO | 000001 |
| 0026 | PRMALS | 000145 |
| 0027 | PRMORG | 000455 |
| 0030 | PRPCOM | 000303 |
| 0031 | PRPNPT | 000076 |
| 0032 | SAVNCR | 000175 |
| 0033 | TURB | 000001 |
| 0034 | VARCOM | 000645 |
| 0035 | WALL | 000226 |
| 0036 | ACCN | 000003 |
| 0037 | ACPK | 000002 |
| 0040 | RETH | 000001 |
| 0041 | RUF | 000022 |
| 0042 | PARTI | 000020 |
| 0043 | LAM | 000001 |

EXTERNAL REFERENCES (BLOCK, NAME )

| 0044 | EQUTL |
| :---: | :---: |
| 0045 | STATE |
| 0046 | LINCER |
| 0047 | TRMBL |
| 0050 | IMONE |
| 0051 | TVCM 1 |
| 0052 | ICOEFF |
| 0053 | TVCCOE |
| 0054 | IONLY |
| 0055 | TVCI |
| 0056 | LIAD |
| 0057 | ABMAX |
| 0060 | RERAY |
| 0061 | RNLCER |
| 0062 | NERR3\$ |

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

| 0001 | 000067 | 10L | 0001 | 000626 | 100L | 0001 | 001141 | 1001L | 0001 | 000163 | 11L | 0001 | 000157 | 12L |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0001 | 000642 | 120 L | 0001 | 001022 | 145 L | 0001 | 000056 | 167G | 0001 | 000113 | 202G | 0001 | 000137 | 216 G |
| 0001 | 000204 | 230G | 0001 | 000206 | 234G | 0001 | 001163 | 244L | 0001 | 000247 | 251G | 0001 | 000345 | 270G |
| 0001 | 000346 | 273G | 0001 | 000025 | 3L | 0001 | 000364 | 302G | 0001 | 000417 | 317G | 0001 | 000426 | 325G |
| 0001 | 000435 | 3316 | 0001 | 000501 | 354G | 0001 | 000504 | 360G | 0001 | 000513 | 364G | 0001 | 000030 | 4L |
| 0001 | 000604 | 417 G | 0001 | 000671 | 441 G | 0001 | 000672 | 444G | 0001 | 001002 | 470G | 0001 | 001033 | 500G |
| 0001 | 001114 | 512G | 0001 | 001117 | 516G | 0001 | 001134 | 526G | 0001 | 000373 | 53L | 0001 | 001173 | 535G |
| 0001 | 000401 | 54L | 0001 | 001274 | 560G | 0001 | 001304 | 565G | 0001 | 001324 | 574 G | 0001 | 000424 | 58L |
| 0001 | 000544 | 60L | 0001 | 001350 | 605G | 0001 | 001204 | 605L | 0001 | 001376 | 614G | 0001 | 001407 | 621G |
| 0001 | 001445 | 6291 | 0001 | 001425 | 630G | 0001 | 001472 | 642G | 0001 | 001474 | 645G | 0001 | 001512 | 654G |
| 0001 | 001520 | 660G | 0001 | 001663 | 6651 | 0001 | 001545 | 671G | 0001 | 001703 | 673L | 0001 | 001720 | 675L |
| 0001 | 002320 | 69 L | 0001 | 001572 | 701G | 0001 | 001617 | 704G | 0001 | 001624 | 710 G | 0001 | 001646 | 717G |


| 0001 |  | 002022 | 740 L | 0001 |  | 002005 | 747G | 0001 |  | 002077 | 760G | 0001 |  | 002121 | 765L | 0001 |  | 002127 | 770 L |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0001 |  | 002163 | 775G | 0001 |  | 002366 | 79L | 0001 |  | 002406 | 850L | 0001 |  | 000612 | 95L | 0001 |  | 002351 | 98L |
| 0001 |  | 002353 | 991 | 0014 | R | 000044 | A | 0000 | R | 000020 | AA | 0000 | R | 000017 | $\triangle C C P$ | 0036 | R | 000000 | ACCPK |
| 0037 | R | 000000 | ACCPK 1 | 0037 | R | 000001 | ACCPK2 | 0042 |  | 000015 | AK | 0034 | R | 000000 | ALPH | 0020 | R | 000000 | ALPHD |
| 0024 | R | 000000 | AM | - 0010 | I | 000000 | ASU | 0013 |  | 000000 | ATA | 0016 | R | 000321 | BA1 | 0016 | R | 001727 | BA2 |
| 0000 | $R$ | 000021 | BB | 0020 | R | 000001 | BETAP | 0020 | R | 000063 | BETAV | 0010 | I | 000003 | BSU | 0004 | R | 000000 | BUMP |
| 0022 |  | 000000 | CASE | 0022 | R | 0000015 | CBAR | 0000 | $R$ | 000022 | CC | 0006 |  | 000000 | CK 1 | 0006 | R | 000006 | CK6 |
| 0012 |  | 000000 | CLNUM | 0024 | R | 000000 | CORAR | 0032 | R | 000000 | CORMA | 0026 |  | 000000 | COSALF | 0042 |  | 000004 | CPART |
| 0007 | R | 000000 | CPFL | 0000 | R | 0000015 | CTE | 0020 | R | 000145 | C1 | 0005 | R | 000005 | C10 | 0005 |  | 000006 | C12 |
| 0005 | R | 000007 | c13 | 0005 |  | 000010 | C14 | 0020 |  | 000146 | C2 | 0020 | R | 000147 | C3M | 0005 | R | 000013 | C32 |
| 0005 | R | 000000 | C5 | 0005 | R | 000001 | C6 | 0005 | R | 000002 | C7 | 0005 | R | 000003 | C8 | 0005 | R | 000004 | C9 |
| 0030 |  | 000255 | DCAPCH | 0024 | R | 035431 | DLPH | 0024 | R | 035441 | DLPK | 0007 |  | 000001 | DPR | 0030 | R | 000000 | DRHOH |
| 0030 | R | 000001 | DRHOK | 0015 | R | 000612 | DRNL | 0011 |  | 000000 | DSIP | 0030 | R | 000254 | DTH | 0024 | R | 035521 | DTHW |
| 0030 | R | 000037 | DTK | 0024 | R | 035522 | DTKW | 0000 | R | 000012 | DUB | 0011 | R | 000062 | DUEDGE | 0000 | R | 000014 | DUM |
| 0041 |  | 000000 | DUMM1 | 0041 |  | 000011 | DUMM.10 | 0041 |  | 000012 | DUMM11 | 0041 |  | 000013 | DUMM 12 | 0041 |  | 000017 | DUMM16 |
| 0041 |  | 000020 | DUMM17 | 0041 |  | 000021 | DUMM 18 | 0042 |  | 000006 | DUMM19 | 0041 |  | 000001 | DUMM2 | 0042 |  | 000011 | DUMM20 |
| 0042 |  | 000012 | DUMM2 1 | 0042 |  | 000010 | DUMM22 | 0042 |  | 000003 | DUMM23 | 0042 |  | 000001 | DUMM24 | 0041 |  | 000002 | DUMM3 |
| 0041 |  | 000003 | DUMM4 | 0041 |  | 000004 | DUMMS | 0041 |  | 000005 | DUMM6 | 0041 |  | 000006 | DUMM7 | 0041 |  | 000007 | DUMM8 |
| 0041 |  | 000010 | DUMM9 | 0032 | R | 000001 | DVNL | 0004 | R | 000001 | EASE | 0015 | R | 000375 | ELMM | 0010 | R | 000007 | EMISC |
| 0010 |  | 000010 | EMIST | 0010 | R | 000011 | EMIV | 0015 | R | 000376 | ENL | 0015 | R | 000571 | ENLM | 0015 | R | 000601 | ENLMM |
| 0012 | R | 0000021 | EPSA | 0014 | R | 0000000 | EQT | 0016 | R | 000000 | ETA | 0034 | R | 0000001 | F | 0015 | R | 000000 | FLE |
| 0035 | R | 000000 | FLUXU | 0041 |  | 000016 | FMF | 0003 |  | 000000 | FR | 0034 | R | 000075 | G | 0007 | R | 000002 | GC |
| 0026 | R | 000144 | GE | 0011 | R | 000063 | GEP | 0015 | R | 000053 | GLE | 0031 | R | 000000 | HB | 0010 | R | 000014 | HCARB |
| 0010 | R | 000015 | HCHAR | 0011 | R | 000064 | HE | 0020 |  | 000231 | HF | 0031 | R | 000017 | HP | 0010 | R | 000020 | HPG |
| 0010 | R | 000021 | HPYG | 0022 | I | 000016 | I | 0021 |  | 000000 | IBODY | 0041 |  | 000015 | ICF | 0032 | I | 000174 | ICORM |
| 0027 |  | 000000 | IDISC | 0015 | 1 | 000602 | IENLM | 0021 | 1 | 000005 | IFLOW | 0021 | I | 000006 | IGUESS | 0021 |  | 000007 | IH |
| 0036 | I | 000001 | ILAM | 0043 | I | 000000 | ILAMIN | 0042 | 1 | 000013 | ILT | 0000 |  | 000077 | INJP\$ | 0042 |  | 000017 | IPART |
| 0022 | I | 000017 | IQ | 0022 | I | 000020 | IS | 0022 |  | 000021 | ISH | 0014 | I | 000016 | ISP | 0010 | 1 | 000025 | ISU |
| 0022 | I | 000022 | ITS | 0021 | I | 000011 | IWALL | 0000 | I | 000010 | IX | 0004 | I | 000002 | 1777 | 0000 | I 0 | 000001 | J |
| 0000 | I | 000013 | JJ | 0021 |  | 000012 | JW | 0000 | I | 000004 | K | 0022 | I | 000023 | KAPPA | 0023 |  | 000000 | KBC |
| 0023 | I | 000001 | KCC | 0000 | I | 000000 | KK | 0022 |  | 000024 | KONRFT | 0023 | I | 000003 | KQ10 | 0023 | I | 000002 | KQ9 |
| 0022 | I | 000025 | KR9 | 0010 | I | 000026 | KS | 0023 |  | 000004 | KSB | 0023 | I | 000005 | KSOL | 0000 | I | 000005 | L |
| 0016 | I | 000126 | LAR | 0003 | I | 001606 | LEF | 0003 | I | 001616 | LEFS | 0000 | I | 000016 | LPI | 0003 | I | 001604 | L2 |
| 0003 | I | 001605 | L3 | 0000 | I | 000002 | M | 0022 | I | 000107 | MATII | 0022 | I | 000110 | MATIU | 0022 | I | 000111 | MAT2I |
| 0000 | I | 000006 | MM | 0003 | I | 001646 | MOA | 0003 | I | 001742 | MOB | 0022 |  | 000112 | MWE | 0000 | I | 000003 | MX |
| 0000 | I | 000007 | N | 0022 | I | 000113 | NAM | 0013 | I | 000432 | NEL | 0022 | 1 | 000114 | NETA | 0022 | 1 | 000115 | NNLEO |
| 0022 |  | 000116 | NON | 0022 | I | 000117 | NRNL | 0022 |  | 000120 | NS | 0022 | I | 000121 | NSP | 0003 | I | 002036 | NSPEC |
| 0022 | I | 000122 | NSPM1 | 0021 | I | 000014 | NTROPY | 0025 | 1 | 000000 | Nū | 0014 |  | 000017 | P | 0042 |  | 000000 | PARTM |
| 0011 | R | 000065 | PE | 0042 |  | 000014 | PF | 0003 | R | 002037 | PIEASE | 0030 | R | 000000 | PREQ | 0031 |  | 000020 | QR |
| 0017 | R | 000007 | QW | 0014 |  | 000350 | R | 0027 |  | 000227 | RADS | 0040 | $R$ | 000000 | RETHMO | 0012 | R | 000041 | RETR |
| 0031 | R | 000037 | RHO | 0011 | R | 000147 | RHOE | 0031 | R | 000056 | RHOP | 0042 |  | 000002 | RHOPA | 0041 |  | 000014 | RK |
| 0027 | R | 000311 | ROKAP | 0042 |  | 000016 | RP | 0027 | R | 000373 | 5 | 0000 | R | 000011 | SFE | 0034 | R | 000152 | SP |
| 0036 |  | 000002 | SPCT | 0015 | R | 000111 | SPLE | 0033 | R | 0000000 | STURB | 0014 | R | 001237 | TC | 0024 | R | 035530 | TCW |
| 0031 | R | 000075 | TP | 0017 | R | 000017 | TPWALL | 0011 | R | 000767 | TTVC | 0011 |  | 000770 | TVCC | 0011 | R | 001052 | UE |
| 0017 | R | 000010 | VJKW | 0014 | R | 001333. | VLNK | 0024 | R | 035531 | VLNKW | 0030 | R | 000264 | VMU | 0011 | R | 001134 | VMUE |
| 0003 | R | 002040 | W | 0017 | R | 000001 | WALLJ | 0017 | R | 000000 | WALEO | 0004 | R | 000003 | WDOT | 0042 |  | 000007 | WF |
| 0042 |  | 000005 | WP | 0020 | R | 000344 | XI | 0006 | R | 000242 | XM |  |  |  |  |  |  |  |  |






| 00463 | 190* | 1 | 1 : ETA (KAPPA))*CBAR) | ANK 4/83 | 000742 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 00465 | 191* |  | IF (ITS .GT. 1) GO TO 145 | ANK 5/83 | 000771 |
| 00467 | 192* |  | DO $140 \mathrm{~K}=1$. NSP |  | 001002 |
| 00472 | 193* | 140 | IF (LEFS (K) .LE. O . AND. LEF (K) .GT. O) EASE $=0.050$ |  | 001002 |
| 00475 | 194* | 145 | $M=2$ | ANK 4/83 | 001022 |
| 00476 | 195* |  | $M M=M A T 1 J-1$ |  | 001023 |
| 00477 | 196* |  | DO $200 \mathrm{I}=1$, NRNL |  | 001033 |
| 00502 | 197* |  | CALL ABMAX (MM-1. ENL (M), ENLM (I), IENLM (I) |  | 001034 |
| 00503 | 198* |  | $\operatorname{IENLM}(\mathrm{I})=\operatorname{IENLM}(\mathrm{I})+1$ |  | 001053 |
| 00504 | 199* |  | $M=M+M M$ |  | 001056 |
| 00505 | 200* | 200 | $M M=$ NETA - 1 | ANK 8/83 | 001061 |
| 00505 | 201* | C | SOLVE REDUCED SET OF EQUATIONS | B05A2070 | 001061 |
| 00507 | 202* |  | IF (IGUESS . LT. O) RETURN | ANK 4/83 | 001066 |
| 00507 | 203* | C | SCRUNTCH DEFINED ROWS OF AM MATRIX TO THE TOP | B05A2090 | 001066 |
| 00511 | 204* |  | DO $240 \mathrm{M}=1$, NAM | B05A2130 | 001074 |
| 00514 | 205* |  | $E N L(M)=E N L(M+1)$ |  | 001114 |
| 00515 | 206* |  | DO $240 \mathrm{~J}=1$, NNLEQ |  | 001117 |
| 00520 | 207* | 240 | $A M(M, J)=A M(M+1, U)$ |  | 001117 |
| 00523 | 208* |  | IF (KQ10 - LE , O) GO TO 1001 | ANK 4/83 | 001126 |
| 00525 | 209* |  | DO $1000 \mathrm{M}=4$, NAM |  | 001134 |
| 00530 | 210* | 1000 | $A M(M, 3)=A M(M, 3)+\operatorname{ENL}(M) / F(1,3)$ |  | 001134 |
| 00530 | 211* | C | THE FOLLOWING ROUTINE REARRANGES COLUMNS OF THE NOW RECTANGULAR | BO5A2250 | 001134 |
| 00530 | 212* |  | AM MATRIX, ACCORDING TO LAR, INVERTS ( AM $\left.\left.^{(I, J), ~} J=2, N A M\right), I=1, N A M\right) ~ A N D$ | B05A2260 | 001134 |
| 00530 | 213* | C | MULTIPLIES THE INVERSE TIMES THE REMAINING COLUMNS OF AM MATRIX | BO5A2270 | 001134 |
| 00530 | 214* | C | AND TIMES THE ENL | B05A2280 | 001134 |
| 00532 | 215* | 1001 | CALL RERAY (NAM, AM, NSP+1, ENL, 1, LAR, IX, 123. EQT, EQT (106), EQT (219). | ANK $4 / 83$ | 001141 |
| 00532 | 216* | 1 | EQT(332), EQT (445)) | ANK 4/83 | 001141 |
| 00532 | 217* | C | TREAT SURFACE OPTIONS IN RNLCER WITH REDUCED NONLINEAR SET |  | 001141 |
| 00533 | 218* | 244 | CALL RNLCER | ANK 4/83 | 001163 |
| 00533 | 219* | C | DETERMINE MAXIMUM NONLINEAR ERRORS | B05A4010 | 001163 |
| 00534 | 220* |  | DO $605 \mathrm{I}=1$. NRNL |  | 001164 |
| 00537 | 221* |  | IF (ABS(ENLM(I)) .GE. ABS(DRNL(I))) GO T0 605 |  | 001173 |
| 00541 | 222* |  | $\operatorname{ENLM}(\mathrm{I})=$ DRNL (I) |  | 001177 |
| 00542 | 223* |  | $\operatorname{IENLM}(\mathrm{I})=1$ |  | 001201 |
| 00543 | 224* | 605 | CONTINUE |  | 001205 |
| 00545 | 225* |  | $S F E=A L P H * A M A X 1(A B S(B E T A P(I S)), 0.10)$ | ANK $7 / 83$ | 001205 |
| 00546 | 226* |  | DUB $=\operatorname{AMAX1}$ (ABS $(\mathrm{G}(\mathrm{NETA}, 1)-\mathrm{G}(1,1)), 1 . \mathrm{OE} 3)$ |  | 001214 |
| 00547 | 227* |  | $\operatorname{ENLM}(1)=\operatorname{ENLM}(1) / S F E$ | - | 001224 |
| 00550 | 228* |  | ENLM (2) $=$ ENLM(2)/DUB |  | 001227 |
| 00551 | 229* |  | CALL ABMAX (NRNL, ENLM, ENLMM, M) |  | 001232 |
| 00552 | 230* |  | ENLMM $=$ ENLMM/10. |  | 001240 |
| 00553 | 231* |  | $\operatorname{ENLM}(1)=\operatorname{ENLM}(1) * S F E$ |  | 001243 |
| 00554 | 232* |  | $\operatorname{ENLM}(2)=\operatorname{ENLM}(2) *$ DUB |  | 001246 |
| 00555 | 233* |  | ELMM $=A B S$ (ELMM) | B05A4160 | 001251 |
| 00556 | 234* |  | ENLMM $=$ ABS (ENLMM) | ANK 4/83 | 001253 |
| 00556 | 235* | C E | EVALUATE NONLINEAR CORRECTIONS FROM THE REDUCED SET | B05A4180 | 001253 |
| 00557 | 236* |  | DO $615 \mathrm{I}=1$, NAM | B05A4190 | 001274 |
| 00562 | 237* |  | $L=L A R(I)$ | B05A4200 | 001274 |
| 00563 | 238* |  | DVNL $(L)=E N L$ (I) | B05A4210 | 001276 |
| 00564 | 239* |  | DO $615 \mathrm{~K}=1$, NRNL | B05A4220 | 001304 |
| 00567 | 240* |  | $J=K+N A M$ | B0544230 | 001304 |
| 00570 | 241* | 615 | $D V N L(L)=$ DVNL $(L)-$ DRNL $(K) *$ AM $(I, J)$ | B05A4240 | 001307 |
| 00573 | 242* |  | DO $620 \mathrm{~K}=1$. NRNL | B05A4250 | 001324 |
| 00576 | 243* |  | $I=N A M+K$ | B05A4260 | 001324 |
| 00577 | 244* |  | $J=\operatorname{LAR}(I)$ | B05A4270 | 001327 |
| 00600 | 245* | 620 D | $\operatorname{DVNL}(J)=$ DRNL (K) | B0544280 | 001332 |
| 00600 | 246* | C-----R | RECYCLE IF ALPH WANTS TO GO NEGATIVE |  | 001332 |
| 00602 | 247* |  | IF (OVNL (1) . GT. -0.90*ALPH) GO TO 629 |  | 001337 |
| 00604 | 248* |  | DO $627 \mathrm{~K}=$ NUL , NSPM1 |  | 001350 |
| 00607 | 249* |  | WALLJ(K) $=$ VJKW (K)*C3M(IS) | ANK $8 / 83$ | 001350 |




END OF COMPILATION
NO DIAGNOSTICS.

```
@SYS$*MSFCFOR$.FOR,WUS OUTPUT
HSA E3 -12/10/84-22:24:00 (39,40)
-8,8
    COMMON/EDGCOM/DSP(53),PE(50),RHOE(350),TE(51),TVCC(50),UE(50),
    $ VMUE(50)
-30
    COMMON/RUF/DUMM1,DUMM2, DUMM3,DUMM4, DUMM5 DUMM6, DUMM7,DUMM8,DUMM9,
    $ DUMM 1O,DUMM11,DUMM12,RK,ICF,FMF,DUMM16,DUMM17,DUMM18
    COMMON/PARTI /PARTM,DUMM24,RHOPA,DUMM23,CPART,WP,DUMM19,WF,
    $ DUMM22, DUMM2O,DUMM21,ILT,PF,AK,RP,IPART
        COMMON /LAM/ ILAMIN
-37
        COMMON/ACCN/ACCPK.ILAM,SPCT
        COMMON/RETH/RETHMO
        DIMENSION DM1(15),DM2(15)
-99,93-
\//\
```

```
            IF(IPART.EQ.1)GO TO 40
        IF(ICF.GT.O.AND.ICF.LT.3)GO TO 40
        GO TO 41
    40 DUMM1=DER(11)*2.
        DUMM2=DER(12
        DUMM3 =S(IS)
        AM=F(NETA, 2)/ALPH*UE(IS)/SQRT (GMR(NETA)/VMW (NETA)*TT(NETA)*49732.)
        REFF=(1.+(GMR(NETA)-1.)/2.*PR(NETA)**.333*AM**2)/
        $ (1.+(GMR(NETA)-1.)/2.*AM**2)
        DO 42 I=1,NETA
        DM1(I)=CPBAR(I )*UCT/UCE
        DM2(I)=TT(I )/UCT
4 2
    CONTINUE
        DZERO=DM1(1)-(DM1(2)-DM1(1))/(DM2(2)-DM2(1))*DM2(1)
        AINT =0.5*(DZERO+DM1(1))*DM2(1)
        DUMMG=AINT
        DO 43 I=2,NETA
        AINT=AINT+O.5*(DM1(I-1)+DM1(I))*(DM2(I)-DM2(I-1))
    43
CONTINUE
        DUMM4 =A INT
        DUMM5 = (DUMM4+(G(NETA, 1)-HB (NETA)/UCE))*REFF
        DUMM7 = RHO (NETA)
        DUMM8 =RHO ( 1)
        DUMM9 = VMU (1)
        DUMM 1O=VMUE (IS)
        OUMM11=PR(NETA)
        DUMM12=UE(IS)
        DUMM12=UE(IS)
        IF(IPART.EQ.1.AND.(ICF.EQ.O.OR.ICF.EQ.3))GO TO 41
        AFACT =0.52
        CALL ROUGH(AFACT)
        CF=DUMM18/2.
        ST =DÜMM16
        WALLQ=ST*(G(NETA,1)-G(1,1))*RHOE(IS)*UE(IS)
        41 CONT INUE
        IF(IPART.EQ. 1)GO TO 45
        GO TO 44
45 IF(ICF.EO.O.OR.ICF.EQ.3)GO TO 46
        DUMM 1=DUMM18
        DUMM2 =DUMM 16
        CONTINUE
        DUMMS=G(NETA,1)
        DUMMG=G(1.1)
```

DUMM2 $1=T T(1) / U C T$
DUMM22 $=$ DUMM 19*VMU (NETA)/UCV/DUMM 11
DUMM23 = (DUMM7*DUMM12*2. *ROKAP (IS)/RAD5)/DUMM 10
PARTM $=(4 . / 3) *.(22 . / 7) *.((R P / 12) * * 3) * R H O P$.
DUMM24 = PARTM*DUMM $12 /((22 . / 7) * .6 . * R P / 12 . * D U M M 10)$
CALL PARTCL
$C F=D U M M 18 / 2$
ST = DUMM16
WALLQ $=S T *(G(N E T A, 1)-G(1,1)) * R H O E(I S) * U E(I S)$

IF (ICF.EQ.O)GO TO 1111
WRITE (6.1009)
FORMAT ( $/, 1 \mathrm{X}, 56(*$, , REMTECH INC. $11-84,56(\%$,$) )$
FORMAT $\left(/, 1 X, 132\left({ }^{\prime *}{ }^{\prime}\right)\right.$
WRITE $(6,1000) \mathrm{ICF}$, RK
1000 FORMAT (/, 2X,'ROUGHNESS MODULE USED - OPTION , I2./
\$ $6 X$, 'EQUIVALENT SAND ROUGHNESS HEIGHT, $R K=, E 10.3$, (FEET)')
IF (ICF.EQ.3)GO TO 1112
RFACT = DUMM 16/DUMM2
IF (DUMM 17.EQ.O.O)WRITE 6.1001 )RFACT
IF (DUMM17.GT.O.O.AND.DUMM17.LT. 1.O)WRITE 6,1002$) R F A C T$
IF (DUMM17. EQ. 1.0) WRITE 6,1003 ) RFACT
1001
1002 FORMAT ( $6 X$,'SMOOTH', $14 X, 20 X$.'ROUGHNESS FACTOR $=$, F7.3)
1003 FORMAT ( $6 X$, ROUGH', $15 X, 20 X$, ROUGHNESS FACTOR $=, F 7.3$ )
WRITE $(6,1008)$ CF, ST, WALLO
1008 FORMAT ( $1 X, \quad$ CF/2 $=1,1$ PE10.3,5X, $\quad$ ST NO. $=1,1$ PE10. $3,5 X$,
\$ 'HEAT FLUX='. 1PE1O.3)
GO TO 111
1112 CONTINUE
IF (ABS (DUMM17) LE. 0.001 )WRITE $(6,1004)$
…...........IF (ABS (DUMM17-1.) LEE.O.OO1)WRITE 6,1005 )
1004 IF (ABS (DUMM17-2.). LE. O.001)WRITE $(6,1006)$
FORMAT ( $6 X$.'SMOOTH')
1006 FORMAT (6X, RKS BEYOND UPPER LIMIT - EQUATION BECOMES INVALID - -
'THEREFORE RKS $=0.0$ WAS USED.')
WRITE (6, 1010)
1111 CONTINUE
IF (IPART.EQ. 1) GO TO 1301
GO TO 1302
1301 WRITE $(6,1009)$
IF(ILT. EQ. 1)WRITE (6, 1303 )RP, AK, PF
1303 FORMAT (/ 2X,'PARTICLE MODULE USED',/ 6X,'LAMINAR FLOW', $5 X$,
\$'PARTICLE SIZE RP=, E1O.3, IN RADIUS', J. $X X$, 'PARTICLE LOADING $=$,
\$ F10.2. 10X, 'PARTICLE FACTOR $=$ ', F10.4)
IF (ILT.EQ.2)WRITE $(6,1305) R P, W P, P F$
1305 FORMAT (/, 2X, 'PARTICLE MODULE USED', $/, 6 X$, 'TURBULENT FLOW', 5X,
\$'PARTICLE SIZE RP=, E1O.3,'IN RADIUS', /, $1 X$, 'PARTICLE LOADING $=0$.
$\$$ F10.2, 10X, 'PARTICLE FACTOR $=$, F F 10.4)
WRITE 6,1008 ) CF,ST,WALLQ
WRITE $(6,1010)$
1302 CONTINUE

IF(ILAMIN.EQ.O.OR.ILAM.EQ.O)GO TO 1211
WRITE (6,1009)
WRITE (6, 1200) SPCT
1200 FORMAT ( $/, 2 X$, RELAMINARIZATION OCCURED',
$\$ 2 X$, DEGREE OF RELAMINARIZATION $=$, E10..3.' PERCENT')
WRITE (6, 1010)
1211 CONT INUE
WRITE (6, 1007 ) RETHMO, ACCP, ACCPK
1007 FQRMAT (/. $1 \times$, $\quad$ RETHMO ACCN PARA ACCN PARA $, 1,1 X$
\$, (EDGE) (WALL),./, 3X, 1P3E 10.3)

SUBROUTINE OUTPUT ENTRY POINT 003775

STORAGE USED: CODE(1) 0O4015; DATA (O) OO1251: BLANK COMMON(2) 000000
COMMON BLOCKS:

| 0003 | AL | 000002 |
| :---: | :---: | :---: |
| 0004 | BLOCOM | 002043 |
| 0005 | COECON | 000317 |
| 0006 | CONSTS | 000010 |
| 0007 | CRBCOM | 000111 |
| 0010 | EDGCOM | 001216 |
| 0011 | EPSCOM | 000040 |
| 0012 | EOPCOM | 002243 |
| 0013 | ETACOM | 000017 |
| 0014 | FLXCOM | 000020 |
| 0015 | HISCOM | 000426 |
| 0016 | HOLLER | 000056 |
| 0017 | INPUTI | 000015 |
| 0020 | INTCOM | 000123 |
| 0021 | INTERI | 000004 |
| 0022 | PRMALS | 000243 |
| 0023 | PRMORG | 000455 |
| 0024 | PRPCOM | 000303 |
| 0025 | PRPERT | 000151 |
| 0026 | PRPIOP | 000016 |
| 0027 | PRPNPT | 000056 |
| 0030 | RFTCOM | 000045 |
| 0031 | RUF | 000022 |
| 0032 | PARTI | 000020 |
| 0033 | LAM | 000001 |
| 0034 | SAHA | 000151 |
| 0035 | SAVOUT | 000021 |
| 0036 | TEMCOM | 000201 |
| 0037 | TURB | 000020 |
| 0040 | UNI COM | 000011 |
| 0041 | VARCOM | 000645 |
| 0042 | WALL | 000454 |
| 0043 | $A C C N$ | 000003 |
| 0044 | RETH | 000001 |

## EXTERNAL REFERENCES (BLOCK, NAME)

| 0045 | ROUGH |
| :---: | :---: |
| 0046 | PARTCL |
| 0047 | REFIT |


| 0050 | ATAN2 |
| :---: | :---: |
| 0051 | COS |
| 0052 | NWDU\$ |
| 0053 | NIO1\$ |
| 0054 | NIO3\$ |
| 0055 | NIO2\$ |
| 0056 | NWBU\$ |
| 0057 | SQRT |
| 0060 | XPRR |
| 0061 | NERR3\$ |

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)


| 0033 | I 000000 | ILAMIN | 0032 | I 000013 | ILT | 0000 |  | 001211 | INJP\$ | 0032 | I | 000017 | IPART | 0034 | I 000063 | IPASS |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0003 | 000000 | IPLOT | 0034 | I 000064 | IRITE | 0020 | I | 000020 | IS | 0020 |  | 000021 | ISH | 0022 | I 000147 | IST |
| 0007 | I 000025 | ISU | 0017 | I 000007 | ITDK | 0017 |  | 000010 | ITH | 0034 |  | 000065 | ITRCNT | 0022 | 000150 | IU |
| 0003 | I 000001 | IUNIT | 0017 | I 000011 | IWALL | 0000 | I | 000060 | J | 0017 |  | 000012 | JW | 0000 | I 000046 | K |
| 0020 | I 000023 | KAPPA | 0030 | I 000036 | KAPPAL | 0030 | I | 000037 | KAPPAT | 0021 |  | 000000 | KBC | 0020 | 10000024 | KONRFT |
| 0021 | I 000003 | KQ1O | 0021 | 1000002 | KQ9 | 0020 |  | 000025 | KR9 | 0007 |  | 000026 | KS | 0030 | I 000040 | KTURB |
| 0004 | 001604 | 12 | 0000 | I 000110 | M | 0016 | I | 000024 | MASS | 0004 | I | 001646 | MOA | 0004 | I 001742 | MOB |
| 0012 | I 000432 | NEL | 0020 | I 000114 | NETA | 0030 | I | 000041 | NETAL | 0030 | 1 | 000042 | NETAT | 0020 | 000115 | NNLEQ |
| 0020 | I 000116 | NON | 0030 | 000043 | NPOINT | 0020 |  | 000117 | NRNL | 0020 | 1 | 000120 | NS | 0034 | 1000066 | NSU |
| 0020 | I 000121 | NSP | 0004 | I 002036 | NSPEC | 0020 | I | 000122 | NSPM 1 | 0017 | I | 000014 | NTROPY | 0000 | 1000043 | NUM |
| 0012 | 000433 | P | 0032 | R 0000000 | PARTM | 0006 | R | 000003 | PATM | 0010 | R | 000065 | PE | 0032 | R 000014 | PF |
| 0006 | 000004 | PI | 0004 | 002037 | PIEASE | 0000 | R | 000104 | PITOT | 0000 | R | 000061 | POUT | 0025 | R 000055 | PR |
| 0000 | 11.000041 | PRES | 0016 | I 000030 | PRESS | 0000 | R | 000053 | QDIFU | 0034 | R | 000067 | QWG | 0022 | 000156 | RADFL |
| 0023 | R 000227 | RADS | 0022 | R 000154 | RAD5 | 0022 | R | 000155 | RADG | 0030 | R | 000044 | RATLIM | 0000 | R 000077 | REFF |
| 0044 | R 000000 | RETHMO | 0016 | I 000034 | REY | 0000 | R | 000103 | RFACT | 0027 | R | 000037 | RHO | 0010 | R 000147 | RHOE |
| 0032 | R 000002 | RHOPA | 0042 | R 000372 | RHOVW | 0031 | R | 000014 | RK | 0023 | R | 000311 | ROKAP | 0032 | R 000016 | RP |
| 0022 | R 000157 | RTM | 0006 | R 000006 | RVAR | 0023 | R | 000373 | S | 0025 | R | 000074 | SC | 0000 | R 000052 | SHEAD |
| 0016 | I 000036 | SHEAR | 0000 | R 000050 | SHFAC | 0006 | R | 000007 | SIPSF | 0041 | R | 000152 | SP | 0043 | R 000002 | SPCT |
| 0000 | R 000102 | ST | 0007 | R 000110 | STEF | 0037 |  | 000000 | STURB | 0022 | R | 000160 | SUMQG | 0016 | I 000040 | TCON |
| 0010 | R 000705 | TE | 0016 | I 000044 | TEMP | 0036 | R | 000153 | THELEM | 0000 | R | 000070 | THENGY | 0000 | R 000071 | THMOM |
| 0016 | 1 1 000046 | THRUST | 0014 | R 000017 | TPWALL | 0025 | R | 000113 | TT | 0037 | R | 000001 | TURPR | 0010 | R 000770 | TVCC |
| 0040 | R 000000 | UCD | 0040 | R 000001 | UCE | 0040 | R | 000002 | UCL | 0040 | R | 000003 | UCM | 0000 | R 000065 | UCMF |
| 0040 | R 0000004 | UCP | 0040 | R 000005 | UCR | 0040 | R | 000006 | UCS | 0040 | R | 000007 | UCT | 0040 | R 000010 | UCV |
| 0010 | R 001052 | UE | 0035 | R 000001 | UKAPPA | 0016 | I | 000050 | VEL | 0016 | I | 000052 | VIS | 0000 | R 000107 | VISC |
| 0014 | R 000010 | VJKW | 0024 | R 000264 | VMU | 0010 | R | 001134 | VMUE | 0025 | R | 000132 | VMW | 0012 | R 001176 | VNU |
| 0004 | R 002040 | W | 0035 | R 000020 | WALLA | 0014 | R | 000001 | WALLU | 0014 | R | 000000 | WALLQ | 0012 | R 002136 | WAT |
| 0032 | 000007 | WF | 0032 | R 0000005 | WP | 0012 | R | 002147 | WTM | 0005 | R | 000247 | XG | 0015 | R 000344 | XI |
| 0005 | R 000242 | XM | 0005 | R 000254 | XSP | 0022 | R | 000161 | XST | 0036 | R | 000162 | Y |  |  |  |








| 01055 | 329* | 1004 | FORMAT (6X, 'SMOOTH') |  | NEWOO2335 |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 01056 | 330* | 1005 | FORMAT (6X, 'ROUGH') |  | NEWOO2335 |  |
| 01057 | 331* | 1006 | FORMAT (6X,'RKS BEYOND UPPER LIMIT - EQUATION BECOMES INVALID - ' ' |  | NEWOO2335 |  |
| 01057 | 332* |  | \$ THEREFORE RKS $=0.0$ WAS USED. ${ }^{\circ}$ ) |  | NEWOO2335 |  |
| 01060 | 333* |  | WRITE (6, 1010) |  | NEWOO2335 |  |
| 01062 | 334* | 1111 | CONTINUE |  | NEWOO2343 |  |
| 01063 | 335* |  | IF (IPART.EQ, 1)GO TO 1301 |  | NEWOO2343 |  |
| 01065 | 336* |  | GO TO 1302 |  | NEWOO2345 |  |
| 01066 | 337* | 1301 | WRITE $(6,1009)$ |  | NEWOO2347 |  |
| 01070 | 338* |  | IF (ILT.EQ. 1 )WRITE (6,1303)RP, AK, PF |  | NEWOO2353 |  |
| 01076 | 339* | 1303 | FORMAT (/ . 2 X, 'PARTICLE MODULE USED'./.6X, LAMINAR FLOW', 5 X , |  | NEWOO2366 |  |
| 01076 | 340* |  | \$ PARTICLE SIZE RP=', E1O.3, IN RADIUS', \%, $1 \times$, PARTICLE LOADING $=$ ', |  | NEWOO2366 |  |
| 01076 | 341* |  | \$ F10.2,10X.'PARTICLE FACTOR $=$ ', F10.4) |  | NEWOO2366 |  |
| 01077 | 342* |  | IF (ILT.EQ. 2 )WRITE (6,1305) RP, WP, PF |  | NEWOO2366 |  |
| 01105 | 343* | 1305 |  |  | NEWOO2401 |  |
| 01105 | 344* |  | \$ PARTICLE SIZE RP=, EIO.3, IN RADIUS\%,\%, $1 \times$, PARTICLE LOADING $=$, |  | NEWOO2401 |  |
| 01105 | 345* |  | \$ F $10.2,10 X,{ }^{\prime}$ PARTICLE $F A C T O R=$, F 10.4 ) |  | NEWOO2401 |  |
| 01106 | 346* |  | WRITE (6, 1008) CF. ST, WALLQ |  | NEWOO2401 |  |
| 01113 | 347* |  | WRITE (6, 1010) |  | NEWOO2411 |  |
| 01115 | 348* | 1302 | CONT INUE |  | NEWOO2417 |  |
| 01116 | 349* |  | IF (ILAMIN.EQ.O.OR. ILAM.EQ.O)GO TO 1211 |  | NEWOO2417 |  |
| 01120 | 350* |  | WRITE (6,1009) |  | NEWOO2426 |  |
| 01122 | 351* |  | WRITE (6, 1200) SPCT |  | NEWOO2433 |  |
| 01125 | 352* | 1200 | FORMAT (/, 2 X , 'RELAMINARIZATION OCCURED', $/$ |  | NEWOO2441 |  |
| 01125 | 353* |  | \$ 2X, 'DEGREE OF RELAMINARIZATION = ', E10.3.' ${ }^{\prime}$ PERCENT') |  | NEWOO2441 |  |
| 01126 | 354* |  | WRITE (6,1010) |  | NEWOO2441 |  |
| 01130 | 355* | 1211 | CONT INUE |  | NEWOO2447 |  |
| 01131 | 356* |  | WRITE (6, 1007 ) RETHMO, ACCP, АССРK |  | NEWOO2447 |  |
| 01136 | 357* | 1007 | FORMAT (/. $1 \times$, ${ }^{\prime}$ RETHMO ACCN PARA ACCN PARA'. $/ 1 \times$, |  | NEWOO2456 |  |
| 01136 | 358* |  | \$\% (EDGE) (WALL) , /, 3X, 1P3E10.3) |  | NEW002456 |  |
| 01137 | 359* |  | IF (IPASS . EQ . 1) GO TO 55 | PLOT | 002456 |  |
| 01141 | 360* |  | $A C C P=1 . O E G * A C C P$ |  | 002461 |  |
| 01141 | 361* | C | STORE ON DRUM FOR PLOTTING: TOTAL HEAT TO WALL, WALL AREA. THRUST | LOSS. | 002461 |  |
| 01141 | 362* | C | ACCELERATION PARAMETER, INVISCID MASS FLOW, AND TOTAL MASS FLOW |  | 002461 |  |
| 01142 | 363* |  | WRITE (3) SUMQG, WALLA, DF, ACCP, THENGY, THMOM | PLOT | 002464 |  |
| 01152 | 364* |  | WRITE (6,6) SHEAR (IUNIT), (ENERGY (IUNIT). K $=1.2$ ) | ANK $8 / 83$ | 002477 |  |
| 01161 | 365* | 6 | FORMAT ( 1 H1, 5X, 'NODAL INFORMATION'//1X, 2HNO, $7 \mathrm{X}, 3 \mathrm{HETA}, 10 \mathrm{X}, 4 \mathrm{HU} / \mathrm{UE}$, |  | 002552 |  |
| 01161 | 366* |  | $18 X, 5 H G A M M A, 8 X, G H S H E A R, A 6,3 X,{ }^{\prime}$ STREAM FUNCTION F', $8 X,{ }^{\prime} \mathrm{FPP}, 12 X$, |  | 002552 |  |
| 01161 | 367* |  | 2 'GP 'A6, 8X,'GPP ', A6) |  | 002552 |  |
| 01162 | 368* |  | DO $183 \mathrm{I}=1$, NETA | B11A 240 | 002552 |  |
| 01165 | 369* |  | DER(1) $=\mathrm{F}(\mathrm{I}, 2) / \mathrm{ALPH}$ |  | 002552 |  |
| 01166 | 370* |  | $\operatorname{DER}(2)=\operatorname{DUDS}(1) / U C S$ |  | 002555 |  |
| 01167 | 371* |  | $D E R(3)=F(1,3) / A L P H * * 2$ |  | 002560 |  |
| 01170 | 372* |  | $\operatorname{DER}(4)=G(1,2) /(A L P H * U C E)$ |  | 002563 |  |
| 01171 | 373* |  | DER(5) $=\mathrm{G}(\mathrm{I}, 3) /($ ALPH**2*UCE) |  | 002566 |  |
| 01171 | 374* | C | STORE ON DRUM FOR PLOTTING: ETA VALUES, VELOCITY RATIO, GAMMA, AND | SHEAR. | 002566 |  |
| 01172 | 375* |  | IF (IPASS . NE. 1) WRITE (4) ETA (I), DER (1),GMR(I), DER(2) | PLOT | 002571 |  |
| 01201 | 376* | 183 |  |  | 002603 |  |
| 01213 | 377* | 12 | FORMAT $(1 \times, 12,3 F 13,7,1$ P5E 18, 7) |  | 002623 |  |
| 01214 | 378* |  | WRITE (6,7) DIST IUNIT), DENS (IUNIT), (ENERGY(IUNIT), K $=1,2$ ) , |  | 002623 |  |
| 01214 | 379* |  | 1 PRES (IUNIT), NUM, NUM |  | 002623 |  |
| 01227 | 380* | 7 | FORMAT (//1X, 2 HNO, $5 X$, 'DISTANCE FROM', $8 X,{ }^{\text {, DENSITY', }} 7 \times$, STATIC ENTHA |  | 002653 |  |
| 01227 | 381* |  | 1LPY'. $4 X$. 'TOTAL ENTHALPY', 6X.'PITOT TUBE', 7 X, 'MACH', 7 X , 'MOLECULAR' |  | 002653 |  |
| 01227 | 382* |  |  |  | 002653 |  |
| 01227 | 383* |  | 3 A6, 5X, 'PRESSURE ', A6, $4 \mathrm{X}, \mathrm{A}, 7 \mathrm{X}$, 'WEIGHT', $7 \mathrm{~T}, \mathrm{AG}$ ) |  | 002653 |  |
| 01230 | 384* |  | DO $184 \mathrm{I}=1$, NETA | B11A 259 | 002653 |  |
| 01233 | 385* |  | $\operatorname{GMR}(1)=\operatorname{ABS}(\operatorname{GMR}(\mathrm{I})$ ) |  | 002657 |  |
| 01234 | 386* |  | $A C H=F(I, 2) / A L P H * U E(I S) / S Q R T(G M R(I) / V M W(I) * T T(I) * G C * R V A R) ~$ | ANK $5 / 83$ | 002661 |  |
| 01235 | 387* |  | DER ( 2 ) $=$ RHO ( 1 )/UCD | EV 10/73 | 002676 |  |
| 01236 | 388* |  | $D X=G M R(I)-1.0$ |  | 002701 |  |



| 01461 | 449* | 204 | $S P(I, 3, K)=S P(1,3, K) * A L P H * * 2$ | 003366 |
| :---: | :---: | :---: | :---: | :---: |
| 01464 | 450* | 2041 | IF (IRITE .EQ. O) GO TO 325 | 003377 |
| 01466 | 451* |  | WRITE $(6,16)$ | 003400 |
| 01470 | 452* | 16 | FORMAT (/2X14HMOLE FRACTIONS.7) B11A 130 | 003405 |
| 01471 | 453* |  | DO $196 \quad \mathrm{~J}=1$, NSPEC $\quad$ B11A 274 | 003405 |
| 01474 | 454* | 196 | WRITE $(6,14) \mathrm{MOA}(J), \operatorname{MOB}(J),(F R(U, I), I=1, N E T A)$ | 003423 |
| 01505 | 455* |  | IF (IWALL .EQ. 4) WRITE (6,17) MOA (ISU), MOB (ISU) ANK 4/83 | 003443 |
| 01512 | 456* | 17 | FORMAT (/4X, SURFACE SPECIES IS , 2A6) | 003460 |
| 01513 | 457* | 325 | WALLO $=$-WALLQ*C3M(IS) $\quad$ ANK 8/83 | 003460 |
| 01514 | 458* |  | IF (NON.LT.O) RETURN | 003463 |
| 01516 | 459* |  | $J=$ NETA - 1 | 003471 |
| 01517 | 460* |  | $\mathrm{M}=\mathrm{KAPPA}-1$ | 003474 |
| 01520 | 461* |  | $K=K A P P A+1$ | 003477 |
| 01521 | 462* |  | NETAL =NETA | 003502 |
| 01522 | 463* |  | KAPPAL = KAPPA | 003504 |
| 01523 | 464* |  | IF (KONRFT.EQ.O) RETURN | 003506 |
| 01525 | 465* |  | IF (KO10.GT. O AND . KTURB . GT . O) GO TO 4019 ANK 4/83 | 003513 |
| 01527 | 466* |  | IF (IS - 1) 4002,4021.4002 | 003527 |
| 01527 | 467* | C | TRANSITION TO TURBULENCE - CHANGE NODE DATA | 003527 |
| 01532 | 468* | 4019 | KTURB $=-1$ | 003534 |
| 01533 | 469* |  | $Y(I)=Y(1) * U C L$ | 003535 |
| 01534 | 470* |  | NETA = NETAT | 003541 |
| 01535 | 471* |  | KAPPA $=$ KAPPAT | 003543 |
| 01536 | 472* |  | DO $4020 \mathrm{I}=1$, NETA | 003554 |
| 01541 | 473* | 4020 | F2FIX $(I)=F 2 F I X T(I)$ | 003554 |
| 01543 | 474* |  | DO 4018 I $=$ NETAL, J | 003562 |
| 01546 | 475* | 4018 | $T T(I+1)=-1.0$. ${ }^{\text {a }}$ (83 | 003562 |
| 01550 | 476* | 4021 | IF (NTROPY . EQ. O) GO TO 4002 ANK 4/83 | 003565 |
| 01550 | 477* | C | SPECIAL ENTROPY QPTION NTROPY $=5$ | 003565 |
| 01552 | 478* |  | DO 4000 I $=1, \mathrm{M}$ | 003566 |
| 01555 | 479* | 4000 | $\operatorname{UKAPPA}(I)=F 2 F I X(I) / F 2 F I X(K A P P A)$ | 003603 |
| 01557 | 480* |  | UKAPPA $(K A P P A)=1.0$ | 003606 |
| 01560 | 481* |  | DO 4001 I $=$ K, J | 003614 |
| 01563 | 482* | 4001 |  | 003614 |
| 01565 | 483* |  | UKAPPA $($ NETA $)=1.0$ | 003620 |
| 01566 | 484* | 4002 | IF (KTURB . NE. - 1 ) GO TO 4022 | 003623 |
| 01570 | 485* |  | $K T U R B=0$ | 003625 |
| 01571 | 486* |  | GO TO 327 | 003626 |
| 01572 | 487* | 4022 | IF (IS .EQ. NS) RETURN | 003630 |
| 01574 | 488* |  | IF (NTROPY . EQ. O) GO TO 4012 ( ${ }^{\text {a }}$ ( $4 / 83$ | 003635 |
| 01574 | 489* | C | SPECIAL ENTROPY OPTION NTROPY $=5$ | 003635 |
| 01576 | 490* |  | DO $4010 \mathrm{I}=1 . \mathrm{M}$ | 003654 |
| 01601 | 491* | 4010 | F2FIX (I) $=$ UKAPPA (I)*F(KAPPA, 2)/ALPH | 003654 |
| 01603 | 492* |  | F2FIX ${ }^{\text {KAPPA }}$ ) $=F(K A P P A, 2) / A L P H$ | 003660 |
| 01604 | 493* |  | DO $4011 \mathrm{I}=\mathrm{K}, \mathrm{J}$ | 003667 |
| 01607 | 494* | 4011 | $F 2 F I X(1)=(F(K A P P A, 2)+(F(N E T A, 2)-F(K A P P A, 2)) * U K A P P A(I)) / A L P H$ ANK $8 / 83$ | 003667 |
| 01611 | 495* |  | $F 2 F I X(N E T A)=F(N E T A, 2) / A L P H$ | 003674 |
| 01612 | 496* | 4012 | IF (IS . EQ: 1) GO TO 327 | 003700 |
| 01614 | 497* |  | $00326 I=2 . J$ | 003702 |
| 01617 | 498* |  | $M=1$ | 003707 |
| 01620 | 499* |  |  | 003711 |
| 01622 | 500* | 326 | $I F(A B S((F) T, 2)-F 2 F I X(I) * A L P H) /(F(M, 2)-F(M-1,2))$ ).GT.RATLIM)GOTO327ANK 8/83 | 003721 |
| 01625 | 501* |  | KONRFT $=1$ | 003736 |
| 01626 | 502* |  | RETURN | 003740 |
| 01627 | 503* | 327 | CALL REFIT | 003744 |
| 01630 | 504* |  | KONRFT $=2$ | 003745 |
| 01631 | 505* |  | RETURN | 003747 |
| 01632 | 506* |  | END ${ }^{\text {B11A }} \mathbf{3 0 7}$ | 004014 |

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@SYS\$*MSFCFOR\$.FOR,WUS TRMBL
HSA E3 -12/10/84-22:24:09 (31,32)


| 0025 | VARCOM | 000645 |
| :--- | :--- | :--- | :--- |
| 0026 | ACPK | 000002 |
| 0027 | ACCN | 000003 |
| 0030 | RETH | 000001 |
| 0031 | RUF | 000022 |
| 0032 | LAM | 000001 |

EXTERNAL REFERENCES (BLOCK. NAME)

| 0033 | LIAD |
| :---: | :---: |
| 0034 | TAYLOR |
| 0035 | ERP |
| 0036 | ERF |
| 0037 | NERR2\$ |
| 0040 | NWDU\$ |
| 0041 | NIO2\$ |
| 0042 | SQRT |
| 0043 | EXP |
| 0044 | XPRR |
| 0045 | TANH |
| 0046 | COSH |
| 0047 | NERR3\$ |

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION. NAME)

| 0001 |  | 002617 | 1000 G | 0001 |  | 000012 | 1001 L | 0001 |  | 000213 | 1002 L | 0001 |  | 001402 | 1003 L | 0001 |  | 004232 | 1004L |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0001 |  | 004245 | 1005L | 0001 |  | 001534 | 103L | 0001 |  | 003114 | 1036G | 0001 |  | 001576 | 104L | 0001 |  | 003454 | 1100G |
| 0001 |  | 003565 | 1122 G | 0001 |  | 003661 | 1136G | 0001 |  | 003716 | 1150G | 0001 |  | 004051 | 1177G | 0001 |  | 004074 | 1210G |
| 0001 |  | 004167 | 1226G | 0001 |  | 004310 | 1255G | 0001 |  | 004434 | 1275G | 0001 |  | 004511 | 1307G | 0001 |  | 004556 | 1317G |
| 0001 |  | 000577 | 15L | 0001 |  | 000074 | $2003 L$ | 0001 |  | 000123 | 2004 L | 0001 |  | 000155 | $2005 L$ | 0001 |  | 002456 | 201 L |
| 0001 |  | 002560 | 202L | 0001 |  | 002553 | 2031 | 0001 |  | 000227 | 247G | 0001 |  | 000264 | 291 | 0001 |  | 001651 | 305L |
| 0001 |  | 001127 | 32 L | 0001 |  | 001702 | 320 L | 0001 |  | 000415 | 321 G | 0001 |  | 000432 | 324G | 0001 |  | 000752 | 331 |
| 0001 |  | 001712 | 330 L | 0001 |  | 001742 | 3314 | 0001 |  | 002022 | 350L | 0001 |  | 002111 | 362L | 0001 |  | 000275 | 391 |
| 0001 |  | 000710 | 400G | 0001 |  | 003773 | 400L | 0001 |  | 004027 | 4011 | 0001 |  | 004063 | 405L | 0001 |  | 003770 | 406 L |
| 0001 |  | 004133 | 415L | 0000 |  | 000360 | 42F | 0001 |  | 000064 | 43L | 0001 |  | 001065 | 4316 | 0000 |  | 000437 | 44F |
| 0000 |  | 000407 | 45 F | 0001 |  | 001215 | 456 G | 0000 |  | 000417 | $46 F$ | 0001 |  | 001220 | 461 G | 0000 |  | 000351 | 47F |
| 0001 |  | 001336 | 4779 | 0000 |  | 000457 | 48F | 0000 |  | 000516 | 49F | 0001 |  | O02143 | 505L | 0001 |  | 002216 | 525L |
| 0001 |  | 002317 | 532 L | 0001 |  | 002664 | 547 L | 0001 |  | 002701 | 550 L | 0001 |  | 003253 | $554 L$ | 0001 |  | 003203 | 555 L |
| 0001 |  | 001656 | 561 G | 0001 |  | 001675 | 573G | 0001 |  | 004252 | 600L | 0001 |  | 001706 | 601G | 0001 |  | 001775 | 616G |
| 0001 |  | 002101 | 636G | 0001 |  | 004600 | 650L | 0001 |  | 002206 | 663G | 0001 |  | 002213 | 670G | 0001 |  | 003257 | 700L |
| 0001 |  | 003650 | 703L | 0001 |  | 002273 | 711G | 0001 |  | 001204 | 75L | 0003 |  | 000000 | A | 0000 | R | 000321 | ABECK |
| 0027 | R | 000000 | ACCPK | 0026 | R | 000000 | $A C C P K 1$ | 0026 | R | 000001 | ACCPK2 | 0000 | R | 000341 | ACEB | 0000 | R | 000325 | ACY |
| 0000 | R | 000311 | AF | 0025 | R | 000000 | ALPH | 0011 |  | 0000000 | ALPHD | 0015 | R | 000000 | AM | 0000 | R | 000301 | BBECK |
| 0011 | R | 000001 | BETAP | 0000 | R | 000312 | BF | 0003 |  | 000003 | C | 0021 | R | 000000 | CAPC | 0000 | R | 000307 | CAPY |
| 0013 |  | 000000 | CASE | 0013 | R | 000015 | CBAR | 0000 | R | 000300 | CBECK | 0000 | R | 000275 | CCEB | 0005 |  | 000000 | CG |
| 0004 |  | 000000 | CK1 | 0004 | R | 000006 | CK6 | 0023 | R | 000000 | CL | 0006 | R | 000000 | CLNUM | 0021 | R | 000017 | CPBAR |
| 0000 | R | 000324 | CRD | 0011 | R | 000145 | C1 | 0003 | R | 000005 | C 10 | 0003 | R | 000007 | C13 | 0011 |  | 000146 | C2 |
| 0003 | R | 000011 | C26 | 0003 | R | 000012 | C28 | 0011 | R | 000147 | C3M | 0003 | R | 000013 | C32 | 0003 | R | 000015 | C53 |
| 0003 | R | 000016 | C56 | 0003 | R | 000002 | C7 | 0003 |  | 000006 | D | 00000 | R | 000320 | DADA | 0000 | R | 000313 | DADPP |
| 0000 | R | 000306 | DADVP | 0000 | R | 000277 | DB | 0020 | R | 000255 | DCAPCH | 0020 | R | 000067 | DCAPCK | 0000 | R | 000000 | DCAPCW |
| 0023 | R | 000001 | DCLNUM | 0000 | R | 000344 | DEACY | 0000 | $R$ | 000304 | DEL | 0023 | R | 000002 | DELCON | 0000 | R | 000007 | DELTA |
| 0023 | R | 000003 | DEPC | 0010 | R | 000017 | DETA | 0006 | R | 000001 | DL | 0000 | R | 000346 | DLDA | 0023 | R | 000004 | DPI |
| 0020 |  | 000256 | DPRH | 0020 | R | 000000 | DRHOH | 0000 | R | 000331 | DRHOI | 0020 | R | 000001 | DRHOK | 0000 | R | 000332 | DUM |
| 0031 |  | 000000 | DUMM 1 | 0031 |  | 000011 | DUMM 10 | 0031 |  | 000012 | DUMM 11 | 0031 |  | 000013 | DUMM 12 | 0031 |  | 000017 | DUMM 16 |
| 0031 | R | 000020 | DUMM17 | 0031 |  | 000021 | DUMM18 | 0031 |  | 000001 | DUMM2 | 0031 |  | 0000002 | DUMM3 | 0031 |  | 000003 | DUMM4 |
| 0031 |  | 000004 | DUMM5 | 0031 |  | 000005 | DUMM6 | 0031 |  | 000006 | DUMM7 | 0031 |  | 000007 | DUMM8 | 0031 |  | 000010 | DUMM9 |
| 0000 | R | 000326 | DUM1 | 0000 | R | 000327 | DUM2 | 0023 | R | 000042 | DVS | 0000 | R | 000071 | DYA | 0003 |  | 000010 | E |
| 0006 | R | 000020 | ELCON | 0007 | R | 000376 | ENL | 0000 | R | 000330 | EPI | 0000 | R | 000322 | EPS | 0006 | R | 000021 | EPSA |


| 0023 | R | 000043 | EPS 1 | 0036 | R | 000000 | ERF | 0035 | R | 000000 | ERP | 0010 | R | 000000 | ETA | 0000 | R | 000345 | EXPA |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0025 | R | 000001 | F | 00007 |  | 000000 | FLE | 0000 | R | 000264 | FM | 0031 |  | 000015 | FMF | 0025 | R | 000075 | G |
| 0022 |  | 0000000 | HB | 0011 | R | 000231 | HF | 0022 | R | 000017 | HP | 0013 | I | 000016 | I | 0012 |  | 000000 | IBODY |
| 0031 | I | 000015 | ICF | 0017 |  | 000000 | IDISC | 0012 | I | 000005 | IFLOW | 0027 | I | 000001 | ILAM | 0032 | I | 000000 | ILAMIN |
| 0000 |  | 000574 | INJP\$ | 0000 | I | 000334 | INK | 0000 | I | 000274 | IPRT | 0013 |  | 000017 | IQ | 0013 | I | 000020 | IS |
| 0013 |  | 000021 | ISH | 0000 | I | 000302 | IWK | 0000 | I | 000335 | J | 0000 | 1 | 000316 | K | 0013 | I | 000023 | KAPPA |
| 0014 |  | 0000000 | KBC | 0013 |  | 000024 | KONRFT | 0014 | I | 000003 | KQ10 | 0013 |  | 000025 | KR9 | 0000 | I | 000315 | L |
| 0013 | I | 000110 | MAT1U | 0013 |  | 000111 | MAT2I | 0000 | I | 000347 | MINK | 0000 | I | 000333 | MPJ | 0013 | I | 000114 | NETA |
| 0013 | I | 000115 | NNLEO | 0013 |  | 000116 | NON | 0013 | I | 000121 | NSP | 0013 | I | 000122 | NSPM1 | 0016 | I | 000000 | NUL |
| 0003 |  | 000014 | 0 | 0000 | R | 000323 | ONK | 0000 | R | 000343 | PCT | 0023 | R | 000044 | PIM | 0023 | R | 000045 | PM |
| 0000 | R | 000317 | PPL | 0021 | R | 000055 | PR | 0006 | R | 000040 | PRT | 0022 |  | 000020 | QR | 0000 | R | 000305 | RC |
| 0023 | R | 000046 | RED | 0030 |  | 000000 | RETHMO | 0006 | R | 000041 | RETR | 0022 | R | 000037 | RHO | 0005 | R | 000147 | RHOE |
| 0022 | R | 000056 | RHOP | 0006 | R | 000042 | RHOVS | 0031 | R | 000014 | RK | 0000 | R | 000342 | RKS | 0017 | R | 000373 | S |
| 0000 | R | 000303 | SALPH | 0006 | R | 000043 | SCT | 0025 | R | 000152 | SP | 0027 | R | 000002 | SPCT | 0000 | R | 000350 | SQPI |
| 0024 | R | 000000 | STURB | 0000 | R | 000336 | TAUW | 0022 | R | 000075 | TP | 0000 | R | 000276 | TPCON | 0023 | R | 000047 | TREF |
| 0005 | R | 000767 | TTVC | 0024 | R | 000001 | TURPR | 0005 |  | 000770 | TVCC | 0005 | R | 001052 | UE | 0000 | R | 000337 | UTAU |
| 0000 | R | 000314 | VA | 0023 | R | 000050 | VINTR | 0020 | R | 000264 | VMU | 0005 | R | 001134 | vmue | 0000 | R | 000340 | VWP |
| 0000 | R | 000270 | XP | 0006 | R | 000044 | YAP | 0000 | R | 000310 | YDI |  |  |  |  |  |  |  |  |




| 00274 | 99* | DEL $=-\mathrm{C3M}(\mathrm{IS}) *$ VMUE (IS) | ANK 8/83 | 000327 |
| :---: | :---: | :---: | :---: | :---: |
| 00275 | 100* | RED $=-$ C3M (IS) *RHOE(IS)*UE (IS) | ANK 8/83 | 000333 |
| 00276 | 101* | RC=RED*CLNUM |  | 000337 |
| 00277 | 102* | $\mathrm{PM}=0.0$ | ANK $5 / 83$ | 000341 |
| 00300 | 103* | EPS $1=0$. |  | 000342 |
| 00301 | 104* | DEPC=0. |  | 000343 |
| 00302 | 105* | RHOVS $=C 1 * F(1,1)+H F(1,5)$ |  | 000344 |
| 00303 | 106* | IF (RC.LT. O.O) GO TO 75 |  | 000350 |
| 00305 | 107* | DADVP $=$ RHOE (IS )/RHO (1) |  | 000361 |
| 00306 | 108* | CAPY $=$ DADVP/RHO (1)*RHOP ( 1 ) |  | 000364 |
| 00307 | 109* | YDI $=0$. |  | 000367 |
| 00310 | 110* | $A F=0.0$ |  | 000370 |
| 00311 | 111* | $B F=0.0$ |  | 000371 |
| 00312 | 112* | $A M(1,1)=0$. |  | 000372 |
| 00313 | 113* | DADPP $=(0.995-C B A R) /(1.0-C B A R)$ |  | 000373 |
| 00314 | 114* | SALPH $=0.0$ |  | 000401 |
| 00315 | 115* | $V A=0.0$ |  | 000402 |
| 00316 | 116* | DVS $=0$. |  | 000403 |
| 00317 | 117** | $L=117$ |  | 000404 |
| 00320 | 118* | DO $66 \mathrm{I}=1$, NETA |  | 000415 |
| 00323 | 119* | DO $3 \mathrm{~K}=1, \mathrm{NSP}$ |  | 000432 |
| 00326 | 120* | DRHOK $(K-1)=A M(L, K+97)$ |  | 000432 |
| 00330 | 121* | PPL $=-$ CAPY |  | 000434 |
| 00331 | 122* | DADA $=$ DADVP |  | 000436 |
| 00332 | 123* | ABECK $=$ YDI |  | 000440 |
| 00333 | 124* | $E P S=B F$ |  | 000442 |
| 00334 | 125* | ONK $=$ RHOE (IS)/RHO(I)**2 |  | 000444 |
| 00335 | 126* | C10 $=$ C7*F(1,2) |  | 000452 |
| 00336 | 127* | $C 56=F(I, 2) / A L P H$ |  | 000455 |
| 00337 | 128* | $C R D=D R H O H * C 10$ |  | 000460 |
| 00340 | 129* | $\mathrm{ACY}=$ - VA |  | 000462 |
| 00341 | 130* | IF (I GE. NETA) GO TO 15 |  | 000464 |
| 00343 | 131* | DADVP $=$ RHOE (IS)/RHO (I+1) |  | 000470 |
| 00344 | 132* | CAPY $=$ DADVP/RHO $(1+1) *$ RHOP ( $1+1)$ |  | 000474 |
| 00345 | 133* | $P P L=P P L+C A P Y$ |  | 000477 |
| 00346 | 134* |  |  | 000501 |
| 00347 | 135* | SALPH $=$ SALPH + YDI |  | 000513 |
| 00350 | 136* | DUM1 $=$ VOI $*(F(I, 3) / D A D A-F(I+1,3) / D A D V P) / 6.0$ |  | 000515 |
| 00351 | 137* | DUM2 $=F($ NETA, 2) $-(F(1,2)+F(I+1,2)) / 2.0$ |  | 000525 |
| 00352 | 138* | DVS $=$ DVS + YDI * (DUM2-DUM1/2.) |  | 000532 |
| 00353 | 139* | $V A=Y D I * * 2$ |  | 000537 |
| 00354 | 140* | $A C Y=A C Y+V A$ |  | 000542 |
| 00355 | 141* | $A F=A F+D E T A(I) / 2 . O *$ (DUM2 - DUM 1) |  | 000544 |
| 00356 | 142* | $\triangle B E C K=A B E C K ~+~ Y D I ~$ |  | 000552 |
| 00357 | 143* | $B F=A L P H * D E L * D E T A(I) / 2.0$ |  | 000555 |
| 00360 | 144* | IF (I . EQ. KAPPA) EPI $=$ BF*DADPP |  | 000562 |
| 00362 | 145* | $I F(I . N E . K A P P A) \quad E P S=E P S+B F$ |  | 000570 |
| 00364 | 146* | DRHOI = - AF*DADA/RHO(I) - F I , 3)/12.0*ACY/RHOE(IS) |  | 000577 |
| 00365 | 147* | IF (CBECK.GT.O.) GO TO 33 |  | 000611 |
| 00367 | 148* | DUM $=A M(L, 98) *$ DRHOI $*$ RC |  | 000613 |
| 00370 | 149* | $A M(1, I+3)=A M(1, I+3)-R C * A B E C K / 2,0+C 7 * D U M * F(1,2)$ |  | 000616 |
| 00371 | 150* | IF (I .LE. 1) AM (1,3) $=A M(1,3)-R C / D A D A * A C Y / 12.0$ |  | 000626 |
| 00373 | 151* | IF (I GT. 1) CALL LIAD (-1, 1, NETA-2+I, -RC/DADA*ACY/12.0) |  | 000640 |
| 00375 | 152* | $A M(1,1)=A M(1,1)-C 7 * D U M * F(1,2) * * 2 / A L P H$ |  | 000663 |
| 00376 | 153* | MP $J=$ MAT $1 \cup+1+1$ |  | 000700 |
| 00377 | 154* | DO $60 \mathrm{~K}=\mathrm{NUL}$, NSPM 1 |  | 000710 |
| 00402 | 155* | IF (K.GT. O) DUM $=$ AM $(L, K+98) * D R H O I * R C$ |  | 000713 |
| 00404 | 156* | IF (I . EQ. NETA) CALL LIAD (K, 1, 1, DUM) |  | 000722 |
| 00406 | 157* | IF (I NE. NETA) AM( 1,MPU) = AM (1,MPU) + DUM |  | 000733 |
| 00410 | 158* | $M P J=M P J+N E T A$ | ANK $8 / 83$ | 000742 |



| 00537 | 219* |  | CRD $=1.0 / \mathrm{PIM}$ |  | 001522 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 00540 | 220* |  | EPS $=-2.0 / P I M * * 2$ |  | 001525 |
| 00541 | 221* |  | GO TO 104 |  | 001532 |
| 00542 | 222* | 103 | $A F=\operatorname{SQRT}(2.0 * D E T A(I-1) / O N K)$ |  | 001534 |
| 00543 | 223* |  | $B F=E R P(A F * P M / 2.0)$ | ANK 5/83 | 001544 |
| 00544 | 224* |  | DADA $=1.0-A F * P M * B F$ | ANK $5 / 83$ | 001553 |
| 00545 | 225* |  | $C R D=E R P(A F * P I M / 2.0)$ |  | 001560 |
| 00546 | 226* |  | $E P S=1.0-A F * P I M * C R D$ |  | 001570 |
| 00547 | 227* | 104 | $B F=B F-E P I * C R D$ |  | 001576 |
| 00550 | 228* |  | DUM1 $=E P I *(A F * C R O-C L) * D E T A(I-1) / 2.0$ |  | 001601 |
| 00551 | 229* |  | $C L=C L * E P I+A F * B F$ |  | 001611 |
| 00552 | 230* |  | $D L(I)=A L P H * E L C O N *(E T A(I)-C L)$ | ANK 5/83 | 001616 |
| 00553 | 231* |  | DUM2 $=$ AF/ONK* $(B F / 2.0+D A D A * A F * P M / 4.0-E P I * E P S * A F * P I M / 4.0)$ | ANK 5/83 | 001623 |
| 00554 | 232* |  | IF (I-2) 305,330,320 |  | 001644 |
| 00557 | 233* | 305 | $D L(1)=0.0$ | ANK 5/83 | 001651 |
| 00560 | 234* |  | DO $307 \mathrm{~J}=1$, NNLEQ |  | 001651 |
| 00563 | 235* | 307 | AM $(2, U)=0$. |  | 001656 |
| 00565 | 236* |  | $\mathrm{CL}=0$. |  | 001657 |
| 00566 | 237* |  | $\operatorname{DPI}(1,2)=\operatorname{CAPC}(1)$ |  | 001660 |
| 00567 | 238* |  | $\operatorname{DPI}(3,1)=F(1,3) * D C A P C H$ |  | 001662 |
| 00570 | 239* |  | IF (NSPMI LE L O) GO TO 350 |  | 001665 |
| 00572 | 240* |  | DO $315 \mathrm{~K}=1$, NSPM 1 |  | 001670 |
| 00575 | 241* | 315 | $\mathrm{DPI}(\mathrm{K}+3,1)=\mathrm{F}(1,3) * \operatorname{DCAPCK}(\mathrm{~K})$ |  | 001675 |
| 00577 | 242* |  | GO TO 350 |  | 001700 |
| 00600 | 243* | 320 | DO $325 \quad J=1$, NNLEQ |  | 001702 |
| 00603 | 244* | 325 | $A M(2, U)=A M(2, U) * E P I$ |  | 001706 |
| 00605 | 245* | 330 | $D U M=-A L P H * E L C O N *(D U M 1+$ + + M $2-E P I * E P S * A F * * 2 / 2.0) * T R E F$ |  | 001712 |
| 00606 | 246* |  | $A M(2,1)=A M(2,1)+(D L(I)-E P I * D L(I-1)) / A L P H$ | ANK $5 / 83$ | 001727 |
| 00607 | 247* |  | $L=1-1$ |  | 001736 |
| 00610 | 248* | 331 | $\operatorname{AM}(2,1)=\operatorname{AM}(2,1)+D P I(1,1) * D U M$ |  | 001742 |
| 00611 | 249* |  | $\operatorname{AM}(2,2)=\operatorname{AM}(2,2)+\operatorname{DPI}(2,1) * \operatorname{DUM}$ |  | 001745 |
| 00612 | 250* |  | $\operatorname{AM}(2,3)=\operatorname{AM}(2,3)+\operatorname{DPI}(1,2) * \operatorname{DUM}$ |  | 001751 |
| 00613 | 251* |  | $A M(2, L+3)=A M(2, L+3)+$ PPI $(2,2) * D U M$ |  | 001755 |
| 00614 | 252* |  | $J=$ MAT $1 J+2$ |  | 001764 |
| 00615 | 253* |  | DO $340 \mathrm{~K}=$ NUL, NSPM 1 |  | 001775 |
| 00620 | 254* |  | $A M(2, J)=A M(2, J)+D P I(K+3,1) * D U M$ |  | 002001 |
| 00621 | 255* |  | $A M(2, J+L-1)=A M(2, J+L-1)+$ DPI $(K+3,2) * D U M$ |  | 002005 |
| 00622 | 256* | 340 | $J=J+$ NETA | ANK $8 / 83$ | 002011 |
| 00624 | 257* |  | IF (L GE . I ) GO TO 400 |  | 002015 |
| 00626 | 258* | 350 | TREF $=$ RED/C26/(2.*CAPC(I)*YAP*PM*YAP*CAPC(I)) | ANK $5 / 83$ | 002022 |
| 00627 | 259* |  | DPI (3,2) $=-\mathrm{PM} / \mathrm{TREF} *(\mathrm{DCAPCH} / \mathrm{CAPC}(1)-\mathrm{DRHOH} /(2 . * R H O(I))$ ) | ANK 5/83 | 002034 |
| 00630 | 260* |  | DPI 2,2$)=$ C10*DPI $(3,2)-$ RHOVS *ALPH |  | 002047 |
| 00631 | 261* |  | DPI $(1,1)=-C 10 * C 56 * D P 1(3,2)-F(1,2) * R H O V S$ |  | 002054 |
| 00632 | 262* |  | $\operatorname{DPI}(2,1)=-A L P H * C 1 * F(1,2)$ |  | 002063 |
| 00633 | 263* |  | IF (NSPM1 . LE O) O) GO TO 362 |  | 002067 |
| 00635 | 264* |  | DO $360 \mathrm{~K}=1$, NSPM1 |  | 002072 |
| 00640 | 265* | 360 | DPI $(K+3,2)=-\mathrm{PM} / \mathrm{TREF} *(\operatorname{DCAPCK}(K) / C A P C(I)-\operatorname{DRHOK}(K) /(2, * R H Q(I)))$ | ANK 5/83 | 002101 |
| 00642 | 266* | 362 | $\mathrm{L}=\mathrm{I}$ |  | 002111 |
| 00643 | 267* |  | DUM $=-$ ALPH*ELCON*(DUM1 - DUM2 + DADA*AF**2/2.O)*TREF |  | 002112 |
| 00644 | 268* |  | IF ( I .LE. 1) RETURN |  | 002126 |
| 00646 | 269* |  | IF (I - NETA) 331,400,400 |  | 002135 |
| 00646 | 270* | C | CEBECI-SMITH AND BECKWITH-BUSHNELL MODELS |  | 002135 |
| 00651 | 271* | 505 | DEL $=-$ C3M (IS)*VMUE(IS) | ANK 8/83 | 002143 |
| 00652 | 272* |  | INK $=1-1$ |  | 002146 |
| 00653 | 273* |  | ONK $=-12.0$ |  | 002151 |
| 00654 | 274* |  | IF (I -GT. 1) GO TO 525 |  | 002153 |
| 00656 | 275* |  | INK $=1$ |  | 002157 |
| 00657 | 276* |  | ONK = ABS (ONK) |  | 002161 |
| 00660 | 277* |  | TAUW $=$ - AMAX $1(\mathrm{C} 28,1 . \mathrm{OE}-4) *$ UE (IS)/ALPH/C3M(IS) | ANK 8/83 | 002163 |
| 00661 | 278* |  | DCAPCW ( 1 ) = DCAPCH |  | 002174 |






END OF COMP ILATION:
NO DIAGNOSTICS.

