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## FINAL REPORT

# SMALL HIGH-SPEED SELF-ACTING SHAFT SEALS FOR LIQUID ROCKET ENGINES

by R. E. Burcham and J. L. Boynton

> Rockwell International Rocketdyne Division

prepared for NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

JULY 1977

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#### SUMMARY

This report covers the design, analysis, fabrication, testing, and evaluation of three self-acting lift pad liquid oxygen face seals and one helium purge floating ring seal with self-acting lift pads on the floating ring ID. The objective of this program was to develop the technology for reliable, 10-hour life, multiple-start seals for use in small, high-speed liquid oxygen turbo-pumps. The seals were designed to operate at 9425 rad/s (90,000 rpm) on a 20-nm OD shaft with 3,102,640 N/m<sup>2</sup>a (450 psia) LOX sealed pressure, 344,737 N/m<sup>2</sup>a (50 psia) helium purge pressure, and 137,895 N/m<sup>2</sup>a (20 psia) maximum drain cavity pressures for 300 starts.

The four seals evaluated were:

- 1. Shrouded Rayleigh step hydrodynamic lift pad LOX face seal with a machined metal bellows secondary seal
- 2. Shrouded Rayleigh step hydrodynamic lift pad LOX face seal with a piston ring secondary seal
- '3. Spiral groove hydrostatic/hydrodynamic LOX face seal with a piston ring secondary.
- 4. Helium purge intermediate floating ring seal with Rayleigh step lift pads on the ring ID

Experimental evaluation of the seals in a turbine driven seal tester is summarized below:

- The Rayleigh step LOX face seal primary element was tested for 11 hours, 40 minutes, including 376 starts, with an average oxygen leakage of 0.762 m<sup>3</sup>/minute (26.9 scfm) at 8063 rad/s (77,000 rpm) and 2,413,165 to 2,771,692 N/m<sup>2</sup>g (350 to 402 psig) sealed pressure. The Rayleigh step concept appears guite feasible for small, high-speed, LOX turbopumps.
- 2. The piston ring secondary LOX seal was tested for 11 hours, 27 minnutes, including 365 starts, and is feasible for small, high-speed liquid oxygen turbopumps.
- 3. The machined metal bellows secondary LOX seal was not feasible due to carbon wear and difficulty of fabrication.
- 4. The spiral groove LOX seal was tested for 11 hours, 43 minutes, including 339 starts with an average leakage of 0.657 m<sup>3</sup>/minute (23.2 scfm) at 8063 rad/s (77,000 rpm) and 2,413,165 to 3,757,903 N/m<sup>2</sup>g (350 to 400 psig) sealed pressure. The spiral groove concept also appears feasible for small, high-speed, oxygen turbopumps.
- 5. Five assemblies of the helium purge intermediate floating ring seal experienced satisfactory performance for 749 tests, totalling 24 hours of operation, demonstrating feasibility of this concept.
- 6. The reverse pumping upstream of the seal was not a reliable method of reducing the pressure at the seal. The pressure drop appeared to be a function of seal cavity through flow.

## INTRODUCTION

Recent system studies of future DoD and NASA reusable vehicles for space maneuvering missions have shown that high-pressure, staged combustion cycle engines offer significant benefit in terms of higher vehicle payload capability. These engines which are in the 44,480 to 111,200 N (10,000 to 25,000 pounds) thrust class, require relatively low flow, high-head turbopumps which are physically smaller and fall outside the design state of the art of rocket turbomachinery. The preliminary designs which have evolved in the studies to date are based on current technology, presupposing a valid extrapolation of this technology to the smaller size. Additionally, and in contrast to past design requirements, reuse encompassing 300 starts and 10 hours time-between-overhauls is envisioned. Thus, designers are confronted with both size and life requirement uncertainties.

Preliminary designs of the oxygen turbopump indicate that shaft speeds up to 9425 rad/s (90,000 rpm) are desirable to achieve low weight and reasonable pump and turbine efficiencies. The shaft used to transmit torque from the turbine to the pump is of such size as to require seals with approximately 20-mm bore diameter. This translates into seal face equivalent rubbing velocities of over 182.88 m/sec (600 ft/sec) and exceeds the state of the art for the conventional rubbing contact seals. Although cryogenic testing has been limited, hydrostatic or hydrodynamic fluid film type seals appear to offer the means to achieve the required multiple starts and extended life capability since the fluid film concept essentially eliminates rubbing contact while maintaining an acceptable leakage rate.

This report documents the work accomplished under NASA Contract NAS3-17769 to develop the technology for reliable, 10-hour life, multiple-start seals for use in small, high-speed liquid oxygen turbopumps. The scope of the program consisted of a review and analysis of primary and intermediate seal designs furnished by NASA. Design and analysis of two alternate primary seal configurations, the spiral groove seal and the Rayleigh step piston ring seal; modification of an existing tester; and experimental evaluation of the NASA seals and the NASA/Rocketdyne selected alternate primary seal using both gaseous nitrogen and liquid oxygen were carried out. Hydrodynamic analysis of the NASA seal designs were provided by NASA-Lewis. Mechanical design, structural, thermal analysis, and dynamic analysis were performed by Rocketdyne. The spiral groove seal was designed and fabricated by Crane Packing Co. to Rocketdyne's specification. The seal tester, designed by Rocketdyne, made maximum use of hardware from an existing tester that was modified for this program. Thermal, stress, including finite element models of seal components, and dynamic analyses of the testor were made and the test program was accomplished at Wyle Laboratories.

#### SEAL DESIGN

A design analysis and detail design were performed on three different selfacting lift pad liquid oxygen face seal designs and one self-acting floating ring helium seal design. Preliminary design layouts were provided by NASA for the Rayleigh step self-acting lift pad bellows LOX seal, piston ring LOX seal, and helium seal. The Rayleigh step lift pad analysis was provided by NASA. The seal detail design was provided by Rocketdyne and the seal suppliers. The spiral groove LOX seal analysis and detail design were provided by Crane Packing Co. in accordance with Rocketdyne specifications.

The LOX seal designs are all interchangeable with each other. The Rayleigh step bellows seal and piston ring seal use the same carbon seal ring and rotating mating ring. The same Rayleigh step lift pad analysis and carbon seal ring deflection analysis were applied to both designs. The same mating ring stress and deflection analysis was applied to all three designs. The spiral groove seal mating ring is the same, except for the spiral grooves. All three designs utilize the same reverse pumping feature to reduce the sealed pressure. The materials are similar on all three seals.

The LOX seals were designed to the following specifications:

```
Fluid:
  Liquid and/or gaseous oxygen
  Gaseous nitrogen
Temperature:
  105 to 145.2 K (-270 to -198 F) oxygen
  294 K (70 F) nitrogen
Pressure:
  137,895 to 3,102,641 N/m<sup>2</sup> (20 to 450 psia)
Speed:
  2617 to 9425 rad/s (25,000 to 90,000 rpm)
Acceleration Rate:
  4188 rad/s/s (40,000 rpm/sec)
Number of Starts:
  300
Operating Life:
  10 hours
Shaft Size:
  20 mm (0.787 in.)
Operating Length:
  0.0254 ±0.0005 m (1.000 ±0.020 in.)
```

The helium seal was designed for  $344,738 \text{ N/m}^2$ a (50 psia) gaseous helium purge at 294.3 K (70 F). The other requirements are the same as for the LOX seal.

RAYLEIGH PAD AND DAM

The Rayleigh step lift pads provide hydrodynamic lift for noncontact operation. A fluid film is developed at the seal face to support the seal ring without rubbing contact. The fluid film thickness or sealing gap is controlled by the hydrodynamic lifting force at the seal face. The lift force decreases for larger gaps and increases for smaller gaps to maintain the desired fluid film thickness. The film thickness is established by balancing the closing forces on the seal ring against the opening forces on the face. The seal ring seeks an equilibrium position where the opening force is equal to the closing force. The operating gap can be adjusted by changing the closing force.

The Rayleigh pad analysis was performed by NASA using a computer program titled NASA Revised Self-Acting Lift Pad Design Program for Gas Film Seals (Ref. 1 ). A summary of the final lift pad geometry is given in Table 1. The final Rayleigh pad carbon seal ring design is shown in Fig. 1.

The lift pad geometry was optimized for gaseous oxygen at 145.2 K (-198 F) to provide margin against rubbing contact due to decreased lift force if the liquid oxygen vaporized in the lift pad area. The lift force at 9425 rad/s (90,000 rpm) and 0.0000025 m (0.0001 in.) film thickness decreases from 146.8 N (33 pounds) for liquid oxygen to 93.4 N (21 pounds) for gaseous oxygen. If a design based on liquid conditions is run with gas, the decreased lift force may result in rubbing contact and seal damage. A gas design seal running in liquid or liquid and gas mixture will operate with a larger film thickness and increased leakage. The increased leakage is more acceptable than rubbing contact.

The relationship of generated lift force and operating film thickness for liquid oxygen, gaseous oxygen, and gaseous nitrogen is shown in Fig. 2 through  $4 \cdot .$  The design condition that results in the minimum lift force determines the maximum seal spring load and pressure closing force. The minimum lift force for a desired operating film thickness of 0.0000025 m (0.0001 in.) is 22.24 N (5 pounds) with gaseous oxygen at the minimum design speed of 2618 rad/s (25,000 rpm). The operating film thickness at the maximum design speed of 9425 rad/s (90,000 rpm) would be approximately 0.0000063 m (0.00025 in.) with gaseous oxygen and 0.000013 m (0.0005 in.) with liquid oxygen using the 22.24 N (5-pound) closing force.

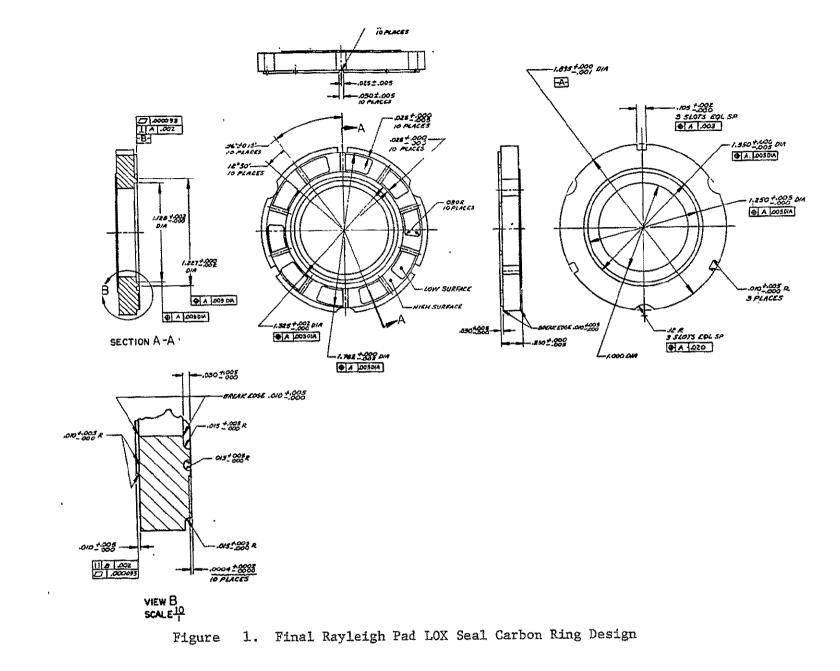
The sealing dam pressure opening force and the closing force due to the pressure acting on the bellows mean effective diameter or the secondary seal diameter were balanced as close as possible for the assumed conditions. The leakage across the sealing dam was assumed to be compressible fluid between parallel sealing surfaces with isentropic entrance conditions and choking (sonic flow) at the exit.

The NASA computer program titled Quasi-One-Dimensional Compressible Flow Across Face Seals and Narrow Slots (Ref. 2 ) was used to analyze the sealing dam opening force and leakage. The predicted average opening pressure at the

### TABLE 1. SUMMARY OF RAYLEIGH PAD LOX SEAL GEOMETRY

Outer Diameter, m (in.)	0.04470 <sup>°</sup> to 0.04415 (1.760 to 1.762)
Inner Diameter, m (in.)	0.03454 to 0.03459 (1.360 to 1.362)
Inside Shoulder, m (in.)	0.00051 to 0.00064 (0.020 to 0.025)
Outside Shoulder, m (in.)	0.00051 to 0.00064 (0.020 to 0.025)
Pad Width (nominal)	0.00406 (0.160)
Total Pad Length (including feed grooves), radians (degrees)	0.628 (36)
Number of Pads	10
Feed Groove, radians (degrees)	0.3840 (22)
Land Length (not including shroud), radians (degrees)	0.1920 (11)
Pad Depth, m (in.)	0.00001 to 0.000015 (0.0004 to 0.0006)
Feed Groove Depth, m (in.)	0.00076 to 0.0089 (0.030 to 0.035)
Feed Groove Radius, m (in.)	0.00038 to 0.00051 (0.015 to 0.020)
Ratio Pad/Land Lengths	2:1
Seal Ring Material	Carbon
Seal Ring Thickness, m (in.)	0.00635 to 0.00648 (0.250 to 0.255)

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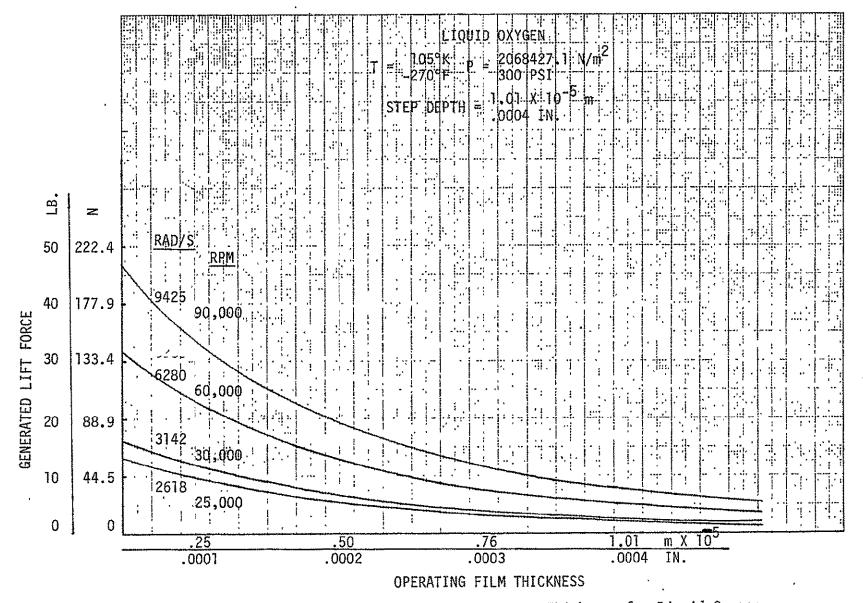


Figure 2. Rayleigh Pad LOX Seal Force vs Film Thickness for Liquid Oxygen

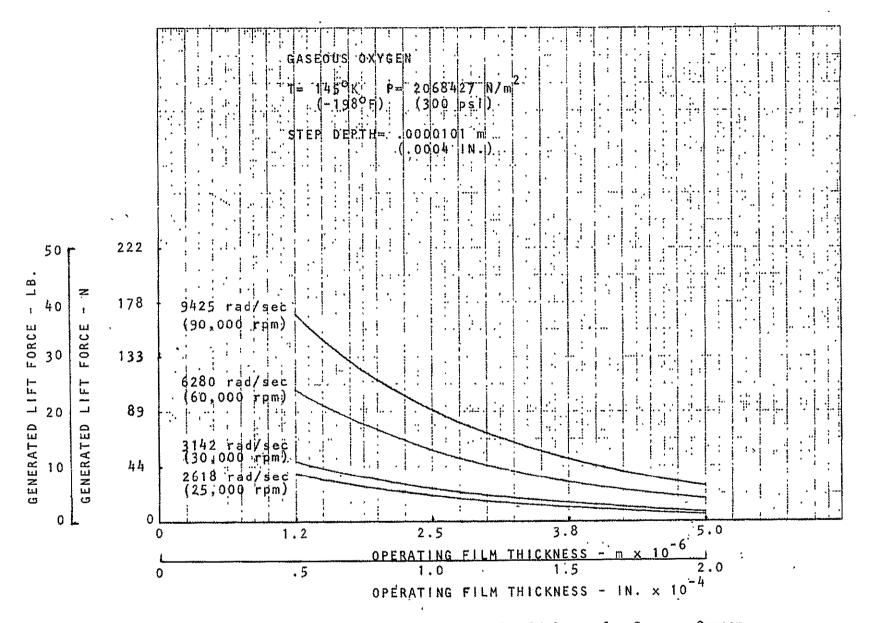
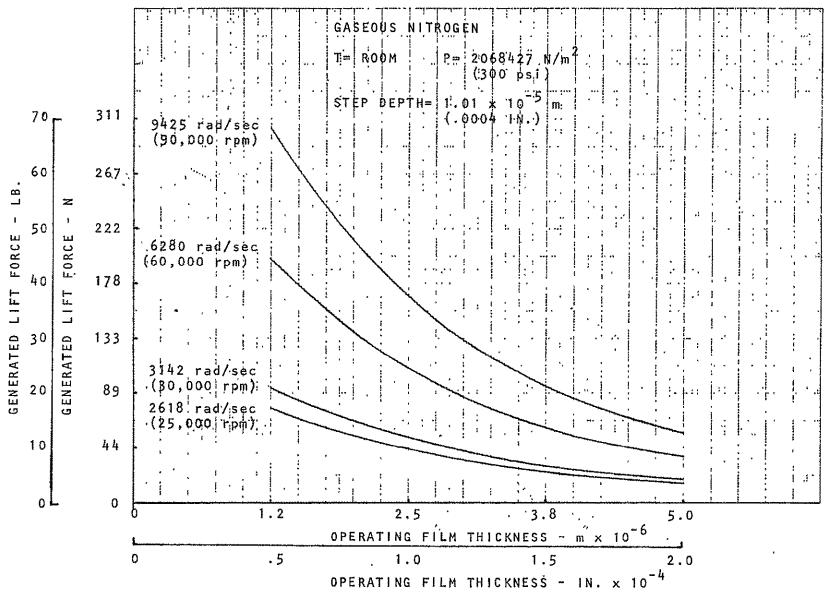
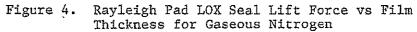


Figure 3. Rayleigh Pad LOX Seal Life Force vs Film Thickness for Gaseous Oxygen





sealing dam is 0.655 to 0.691 (pressure profile factor or force bar) of the pressure differential at the nominal pressure ratio of 20 (assumes 2,757,903 N/m<sup>2</sup>a (400 psia) upstream and 137,895 N/m<sup>2</sup>a (20 psia) downstream for the film thickness of 0.0000025 to 0.00001 m (0.0001 to 0.0004 in.). The variation within the expected pressure ratio range of 15 to 30 is 0.647 to 0.696. The relationship of force bar and pressure ratio is shown in Fig. 5.

The sealing dam dimensions were selected to provide a 0.7 balance ratio (closing area/dam area). The seal is pressure balanced when the balance ratio is equal to the sealing dam pressure profile factor. The design provides a slightly positive pressure closing force to allow for variations in fluid condition and sealing surface geometry.

The predicted sealing dam leakage rate as a function of operating film thickness and speed at pressures of 2,068,427 N/m<sup>2</sup>a (300 psia) and 3,102,641 N/m<sup>2</sup>a (450 psia) for gaseous oxygen and liquid oxygen is shown in Fig. 5 through 8. The liquid leakage tends to decrease at higher speed due to outward viscous pumping which opposes the inward leakage. The gas leakage is not a function of speed. The static leakage at the lapped joint dam between the seal ring and bellows end plate is not included in the sealing dam leakage. The lapped joint leakage is expected to be negligible. The bellows provides a leak-free secondary sealing element.

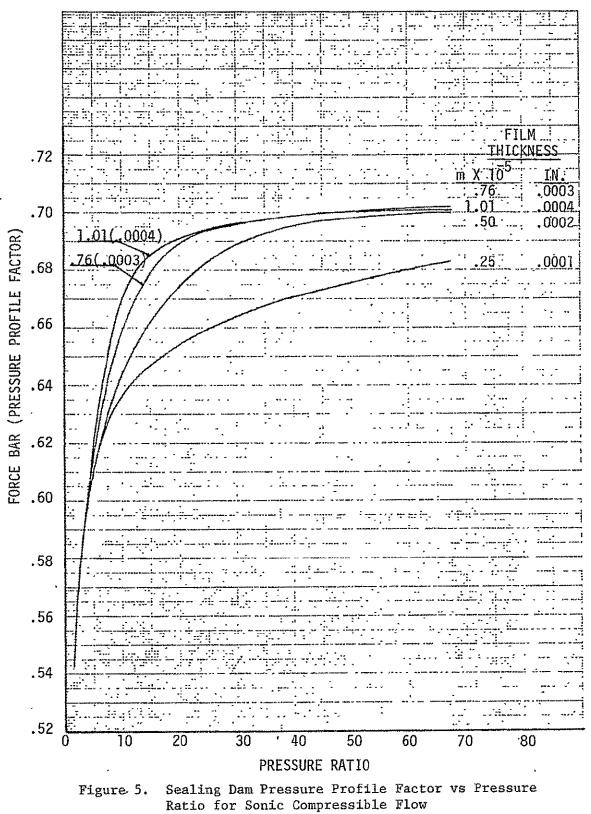
The heat loss in the lift pad area due to viscous shear of the sealed fluid is shown in Fig. 9 for liquid oxygen and Fig. 10 for gaseous oxygen. The heat loss at the sealing dam is shown in Fig. 11 and 12. It is expected that the heat loss to the sealed fluid will vaporize most of the liquid oxygen in the lift pad area.

### **REVERSE PUMPING**

The effects of dynamic pumping on the tester rotating mating ring were investigated as a method of reducing the sealed pressure requirement. Dynamic pumping, on the rotor faces affects the radial face pressure distribution, shaft axial thrust, and heat added to the fluid. The analysis indicates the possibility of reducing the pressure at the LOX seal to 2,068,427 N/m<sup>2</sup>a (300 psia) for a cavity pressure of 3,102,641 N/m<sup>2</sup>a (450 psia) at 9425 rad/s (90,000 rpm) using reverse pumping over the rotating mating ring face.

Reverse pumping results from pumping in the gap between the rotating mating ring and the stationary housing as shown in Fig. 13. The lower pressure at the seal reduces the structural requirements of the seal components and the seal leakage. The use of reverse pumping makes the machined metal bellows design feasible by allowing thinner bellows plates and lower spring rate bellows designs.

Analysis was performed on the reverse pumping concept to determine the mating ring geometry and the seal cavity coolant flowrate to achieve the desired radial pressure gradient. The analysis considered the friction heat addition, fluid temperature increase, head drop, density change, and pressure change versus geometry and coolant flowrate. The friction heat addition for the other



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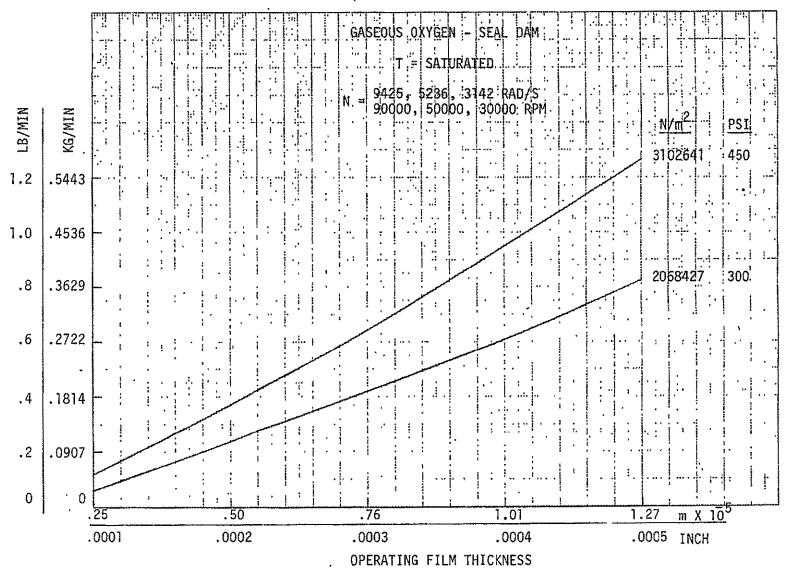
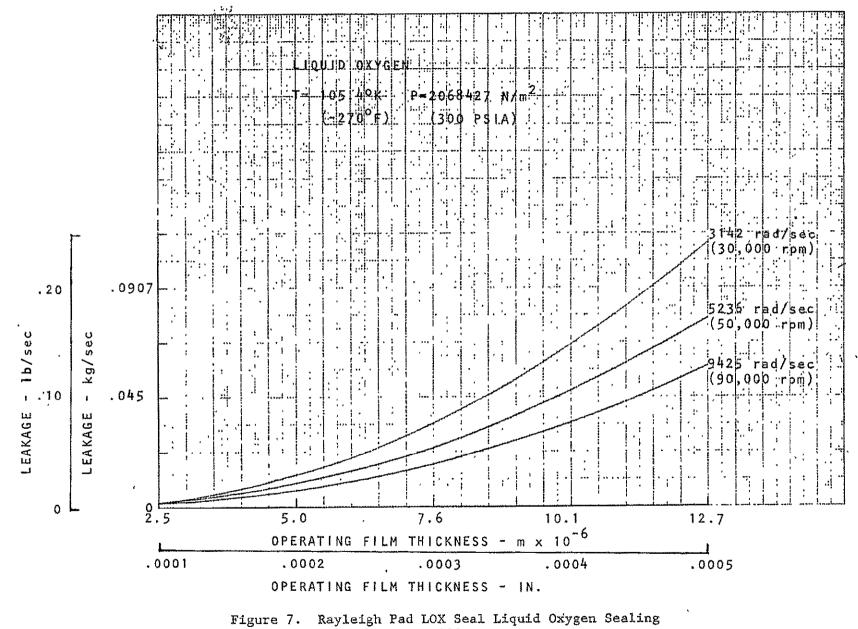


Figure 6. Rayleigh Pad LOX Seal Gaseous Oxygen Sealing Dam Leakage at 300 psia and 450 psia



Dam Leakage at 300 psia

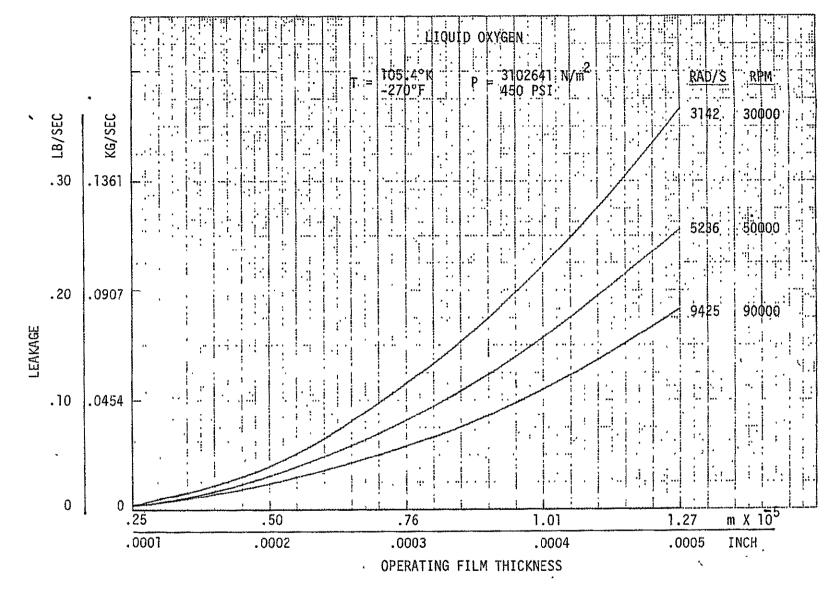


Figure 8. Rayleigh Pad LOX Seal Liquid Oxygen Sealing Dam Leakage at 3,102,641 N/m<sup>2</sup> (450 psia)

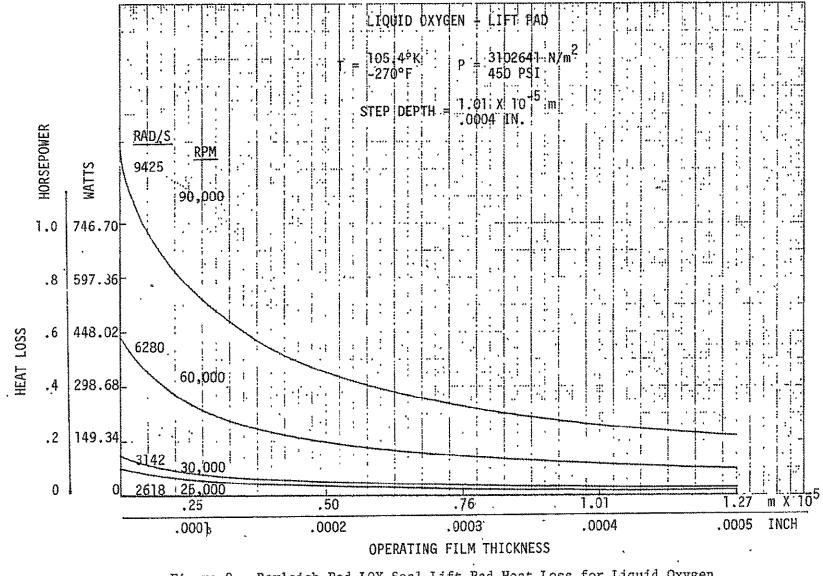


Figure 9. Rayleigh Pad LOX Seal Lift Pad Heat Loss for Liquid Oxygen

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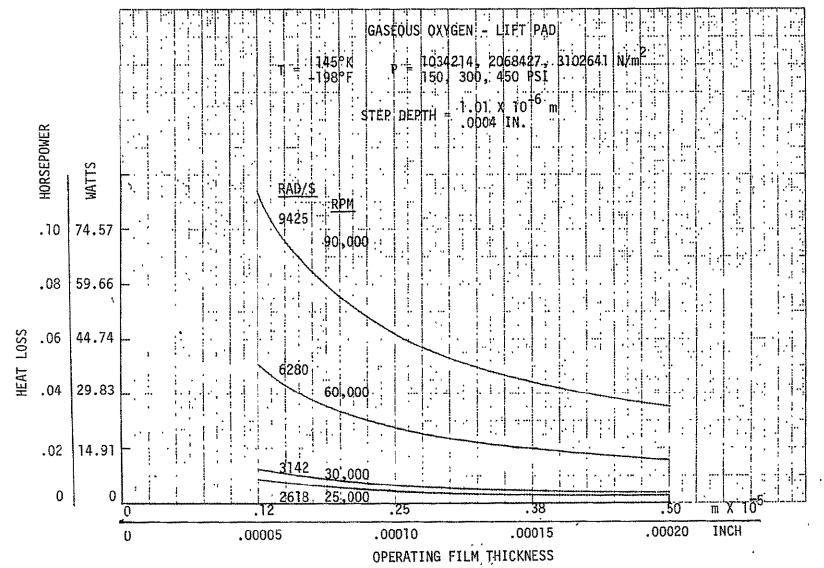


Figure 10. Rayleigh Pad LOX Seal Lift Pad Heat Loss for Gaseous Oxygen

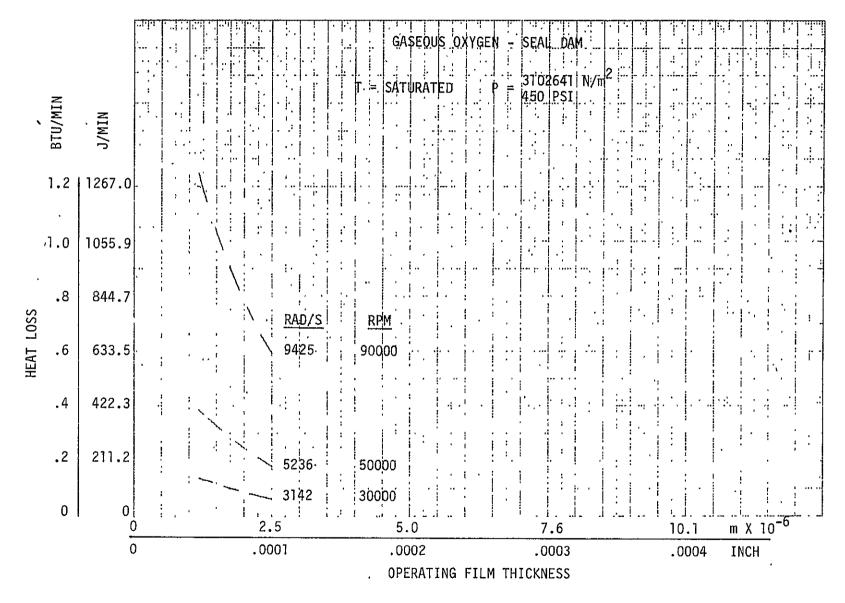
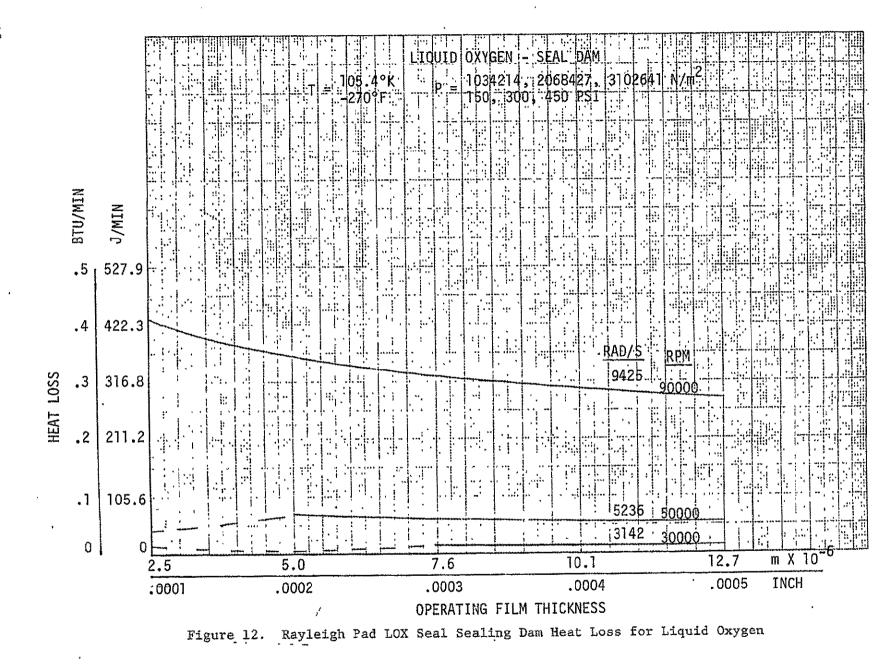


Figure 11. Rayleigh Pad LOX Seal Sealing Dam Heat Loss for Gaseous Oxygen



# CONCEPT: USE REVERSE PUMPING TO REDUCE SEALED PRESSURE TO VAPOR PRESSURE

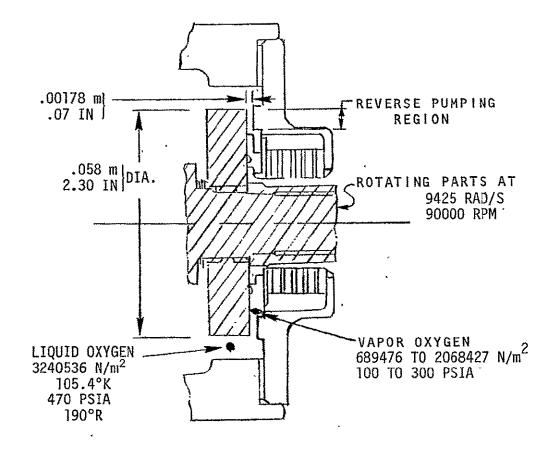


Figure 13. Reverse Pumping Configuration

rotating surfaces in the sealed fluid cavity was calculated to determine the total coolant flowrate. The computer program for the analysis was written for the GE 440 Timeshare computer using an NBS oxygen fluid property subroutine. The reverse pumping mating ring is essentially the same for both the Rayleigh step LOX seal and the spiral groove LOX seal design.

The pressure drop across the reverse pumping element was estimated for liquid oxygen as a function of speed. The upstream cavity pressure was used as the sealed pressure for gaseous nitrogen testing since no significant reverse pumping occurs with the low density gas. A pressure port was added in the oxygen seal housing to measure the actual pressure at the seal.

Analysis indicated that, at 9425 rad/s (90,000 rpm), increasing the face seal rotating mating ring diameter to 0.058 m (2.3 in.), and with an axial gap of 0.00178 m (0.07 in.), would result in reverse pumping of the fluid from the 2,102,641 N/m<sup>2</sup>a (450 psia) cavity pressure to the vapor pressure of 689,476 N/m<sup>2</sup>a (100 psia). The 0.058 m (2.3 in.) diameter mating ring at 9425 rad/s (90,000 rpm) with 0.1814 kg/s (0.4 lb/sec) coolant flow results in 96,266 N/m<sup>2</sup>a (140 psia) oxygen vapor at the LOX seal carbon ring for 3,240,536 N/m<sup>2</sup>a (470 psia) LOX sealed pressure. The design pressure range for seal components is 689,476 N/m<sup>2</sup>a (100 psia) operational minimum and 2,068,427 N/m<sup>2</sup>a (300 psia) structural maximum operating conditions.

CARBON SEAL RING

Preliminary force and moment balances indicated the oxygen face seal carbon ring thickness should be increased to reduce ring deflection. The ring thickness was increased from 0.0038 m (0.150 in.) to 0.00635 m (0.250 in.) which increased section stiffness by a factor of 6.6.

The pressure distribution and axial forces on the carbon ring with reverse pumping is shown in Fig. 14. A seal face pressure of  $896,318 \text{ N/m}^2$  (130 psi) was used for structural analysis. The lapped joint diameter was computed to balance the bending moments on the carbon seal ring.

### MATING RING

The LOX seal mating ring (Fig. 15) was designed to maintain flatness at operating conditions by isolating the ring from the shaft clamping forces and by balancing the bending moments as nearly as possible. The ring mounting configuration is shown in Fig. 16. The static seal provides a 1334 N (300 pound) clamping force that is not a function of the shaft axial stackup load.

Seal face flatness was achieved by finite element analysis using the force and pressure distribution shown in Fig. 17.

Results indicated a 0.785 rad (45 degree) chamfer 0.0023 m (0.090 in.) wide on the non-seal side outer corner would maintain seal face flatness at 9425 rad/s (90,000 rpm) considering pressure-induced forces and Poisson's effect for rotation (Fig. 18).

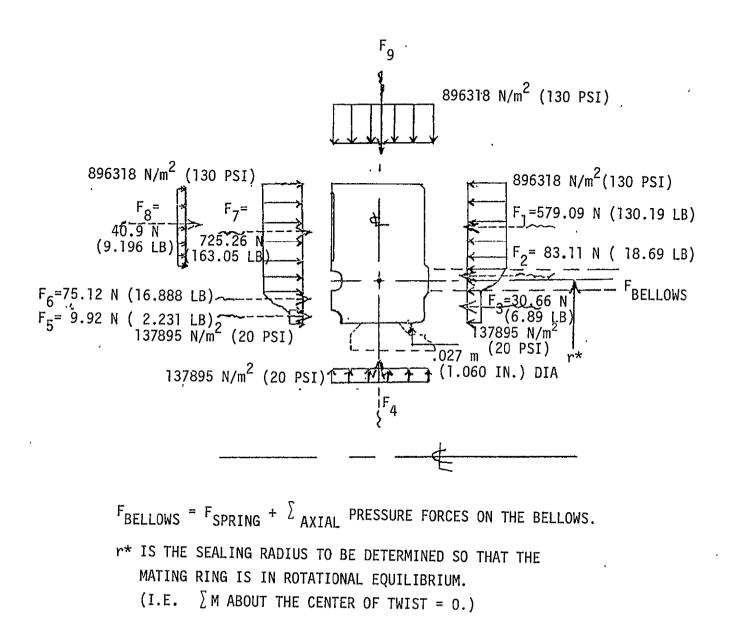


Figure 14. LOX Seal Carbon Seal Ring Pressure Distribution and Axial Forces

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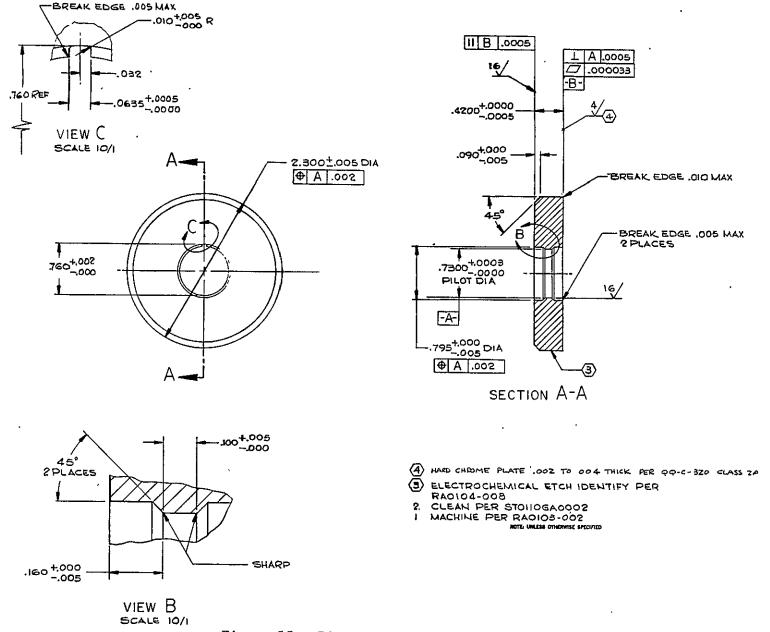


Figure 15. Ring, Mating, LOX Seal

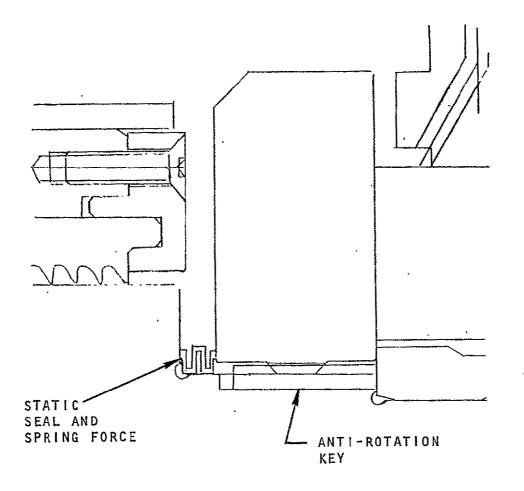
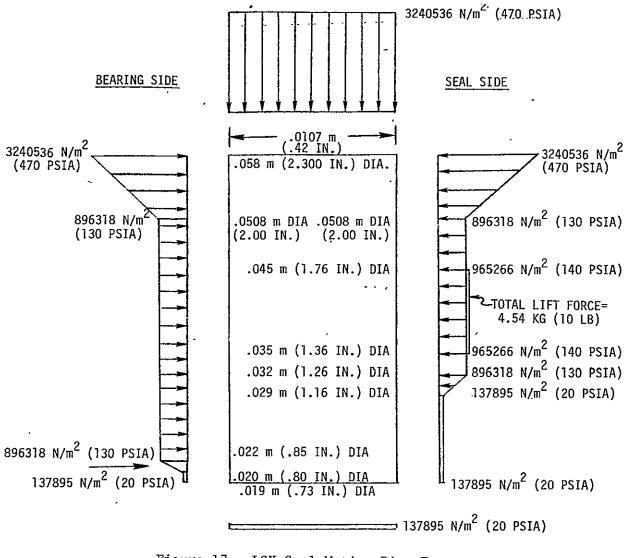
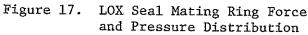
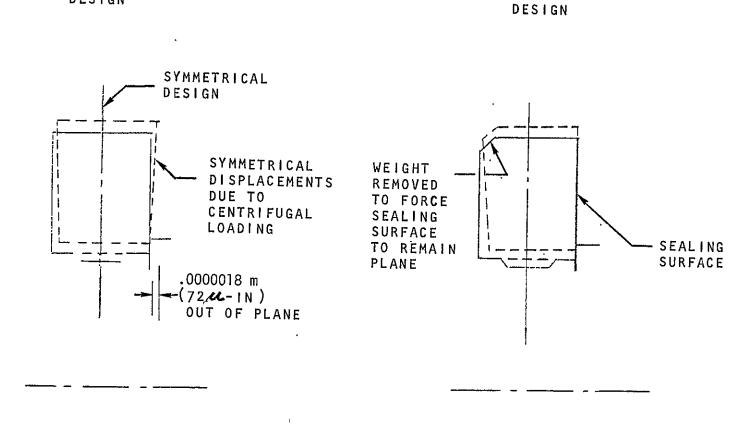


Figure 16. LOX Seal Mating Ring Mounting Configuration







FINAL

Figure 18. LOX Seal Mating Ring Modification to Maintain Flatness

PRELIMINARY

DESIGN

The mating ring OD was increased from 0.0045 m (1.76 in.) to 0.058 m (2.30 in.) diameter to provide a reverse pumping region upstream of the seal.

A narrow pilot was used at the base of the mating ring to center the ring on the shaft and to isolate the ring from any distortions in the shaft.

The mating ring material is K-monel for strength and oxidation resistance. Hard chrome plate was applied over the sealing face for wear resistance.

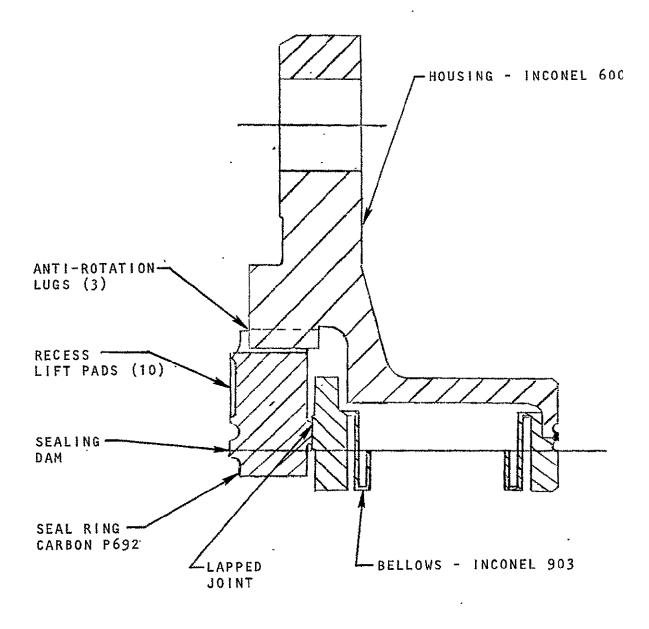
### RAYLEIGH PAD BELLOWS LOX SEAL

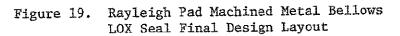
The Rayleigh pad bellows LOX seal (Fig. 19) consists of a face-type carbon seal ring with shrouded recess step hydrodynamic lift pads and a machined metal bellows secondary element. The machined metal bellows provides both the spring force required to seat the seal ring and a positive secondary sealing element between the seal ring and the housing. The bellows loads the carbon seal ring through a lapped joint to provide a static seal. The total closing force consists of the spring load in the bellows and the pressure load acting on the bellows mean effective diameter. The machined bellows provides a nearly constant effective diameter for varying pressure differential to maintain a consistent pressure closing force.

The carbon seal ring is free floating with a radial pilot and three antirotation lugs at the outside diameter. The floating seal ring provides for thermal contraction differentials to minimize distortion. The seal face has 10 recess pads to provide hydrodynamic lift and a continuous dam for sealing. The back side of the seal ring has a lapped joint dam to provide a static seal at the bellows end plate. The friction at the housing pilot and antirotation lugs provides vibration damping for the seal ring to prevent bellows vibration.

The NASA-provided bellows design (Fig. 20 ) was modified as follows:

- 1. Housing changed from press fit capsule to bolted flange for improved installation and static sealing.
- Seal ring pilot moved to outside diameter to eliminate excessive clearance increase at cryogenic (90.38 K; -297 F) temperature due to thermal contraction differentials.
- 3. Seal ring pilot and antirotation lugs moved from the bellows end plate to the housing for vibration damping.
- 4. The carbon seal ring width was increased for additional stiffness to minimize distortion.
- 5. The bellows length was increased to lower the spring rate.
- 6. The mating ring mounting was changed from a clamped sleeve to a spring-loaded floating ring to minimize distortion.
- 7. The mating ring was modified to provide reverse pumping for reduced sealing pressure.
- The seal ring material was changed from carbon PO3N to carbon P692 for improved wear resistance in LOX and to minimize edge chipping.





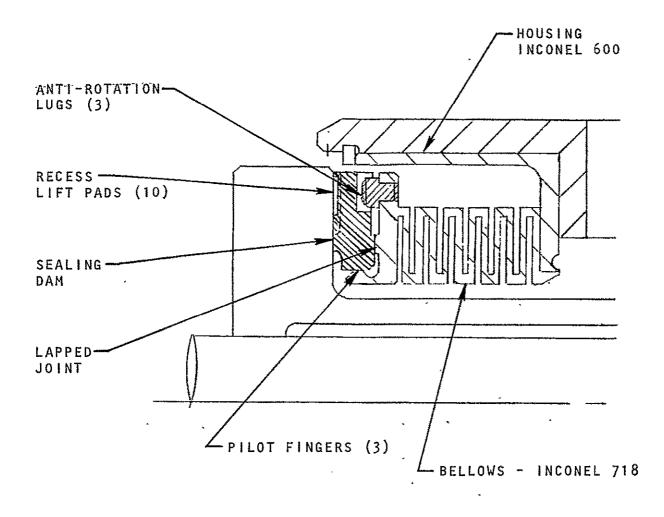


Figure 20. NASA-Provided Bellows LOX Seal Design Layout

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9. The bellows material was changed from Inconel 718 to Inconel 903 for improved machining.

The bellows span, plate thickness and number of convolutions were analyzed to satisfy the stress and spring rate requirements. The pressure stress is proportional to the ratio of the span over the plate thickness  $(s/t)^2$ . The spring rate per convolution is proportional to the inverse of the same ratio  $(t/s)^3$ . Therefore, increasing the plate thickness decreases the pressure stress and increases the spring rate per convolution. The total spring rate is reduced by increasing the number of convolutions. The number of convolutions is limited by the available space.

The desired bellows spring rate was established at 17,512 N/m (100 lb/in.) by the design spring force of 22.2 N (5 pounds) and bellows compression of 0.0013 m (0.050 in.); however, analysis by the bellows manufacturer indicated that the minimum spring rate for the 3,102,641 N/m<sup>2</sup>a (450 psia) pressure requirement was 48,157 N/m (275 lb/in.) using Inconel 718 material. The öther higher strength maraging steel allows (Vascomax 300) were not satisfactory due to low temperature brittleness, notch sensitivity and corrosion.

A review of alternate designs indicated that a significant spring rate increase would result if the design was changed to allow the use of Inconel 718. A comparison of different designs is given below:

Design	Plate Thickness, m (in.)	Span, m (in.)	Stress, N/cm <sup>2</sup> (psi)	Spring Rate, N/m (Ib/in.)		
Present	0.000165 (0.0065)	0.00495 (0.195)	99974 (145,000)	20489 (117)		
Alternate No. l	0.000203 (0.008)	0.00495 (0.195)	74119 (107,500)	38189 (218		
Alternate No. 2	0.000229 (0.009)	0.00495 (0.195)	63432 (920,000)	49016 (280)		
Alternate No. 3	0.000254 (0.010)	0.00495 (0.195)	61639 (864,00)	74409 (425)		
Alternate No. 4	0.000172 (0.0068)	0.00401 (0.158)	74808 (108,500)	43780 (250)		

The bellows manufacturer indicated that it would be necessary either to increase the plate thickness or decrease the span to provide sufficient convolution rigidity for grinding of the Inconel 718 material, due to the large spanto-plate thickness ratio and the difficulty of grinding. The 0.000165 m (0.0065 in.) thick plates do not provide sufficient support for the 0.00406 m (0.016 in.) width grinding disks which are used to machine the convolution grooves.

Analysis indicated that the spring rate could be decreased if the test pressure requirements could be changed from  $3,102,641 \text{ N/m}^2\text{a}$  (450 psia) to 2,068,427 N/m<sup>2</sup>a (300 psia.) Analysis of the turbopump requirements indicated 3,102641  $N/m^2a$  (450 psia) was required in the seal cavity. Incorporating the reverse pumping element upstream of the seal reduced the pressure at the seal to 2,068,427 N/m<sup>2</sup>a (300 psia) at operating speed, making the machined metal bellows feasible

Provisions were made in the housing design for a 0.00178 m (0.070 in.) axial gap between the mating ring face and the housing face for the reverse pumping element. Provisions were also made for a pressure measurement at the seal components downstream of the reverse pumping element.

A review of alternate materials indicated that Inconel 903 would satisfy the new design requirements and was potentially more machinable than Inconel 718. One satisfactory Inconel 903 machined bellows was completed by Hydrodyne. The following design changes were made to allow machining the bellows:

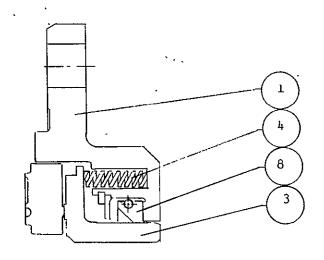
- Plate thickness changed from 0.000165 m (0.0065 in.) to 0.000203 m (0.008 in.)
- Bellows inside diameter changed from 0.0231 m (0.910 in.) to 0.0245 m (0.963 in.)
- 3. Bellows outside diameter changed from 0.0356 m (1.400 in.) to 0.0342 m (1.347 in.)
- 4. Bellows span changed from 0.00622 m (0.245 in.) to 0.00488 m (0.192 in.)
- 5. Bellows mean effective diameter (MED) remains the same at 0.02934 m (1.155 in.)
- 6. Spring rate changed from 20,489 N/m (117 1b/in.) to 35,039 ±10% N/m (200 ±10% 1b/in.) (First part measured 31,171 N/m (178 1b/in.) at Hydrodyne.)

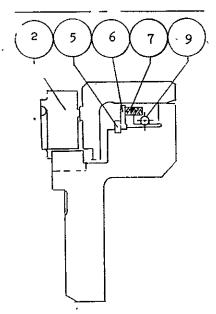
The second bellows seal was delayed due to machining difficulties at Hydrodyne. It was agreed to cancel requirements for the second bellows LOX seal due to machining difficulties. It was concluded that the design is not feasible for consistent manufacture even with the changes made to allow machining the first bellows, and requirements were cancelled. A total of 12 bellows was scrapped in an attempt to make the second bellows seal.

## RAYLEIGH PAD PISTON RING LOX SEAL

The final design (Stein Seal Company) Rayleigh pad piston ring LOX seal (Fig. 21) consists of a face-type carbon seal ring with shrouded recess step hydrodynamic lift pads (same carbon ring as bellows seal) and a pressure-balanced segmented carbon piston ring secondary element. The seal ring is loaded through a pilot ring with compression coil springs.

The carbon seal ring is free floating with a radial pilot and three antirotation lugs at the outside diameter. The back side of the seal ring has a lapped joint dam to provide a static seal to the pilot ring.





ITEM	REQD	PART NO.	DESCRIPTION	MATL.	MATL. SPEC.
9.	1	SSCY 5097-9	SPRING, EXTENSION	INCL X-750	AMS 5698 OR AMS 5699
8	l	SSCY 5097-8	RING, SEAL	CARBON-GRAPHITE	PURE CARBON P5 NR2
· 7	4	SSCY 5097-7.	SPRING, COMPRESSION	INCL X-750	ams 5698 or ams 5699
6	l	SSCY 5097-6	PLATE, BACK	SST	TYPE 302
5	1	SSCY 5097-5	RING, RETAINING	SST	TYPE 302
4	12	. SSCY 5097-4	SPRING, COMPRESSION	SST	TYPE 302
3	l	SSCY 5097-3	RING, PILOT	INCL 718	AMS 5664
2	l	SSCY 5097-2	WAFER	CARBON-GRAPHITE	PURE CARBON GR P692
1	l	SSCY 5097-1	FLANGE	INCL 718	AMS 5664

Figure 21. Rayleigh Pad Piston Ring LOX Seal Final Design (Stein Seal Company)

The NASA-provided composite piston ring LOX seal design is shown in Fig. 22. The design was modified to use the same carbon seal ring as the bellows LOX seal. The composite piston ring was changed to a Stein Seal Company-designed segmented-carbon ring for more effective sealing.

The seal spring load was adjusted by changing the number of springs. The piston ring diameter was established to maintain a 0.7 pressure balance ratio (closing area/dam area) using the same carbon as the bellows LOX seal.

## SPIRAL GROOVE PISTON RING LOX SEAL

The spiral groove piston ring LOX seal was designed by Crane Packing Company in accordance with Rocketdyne specifications (Fig. 23). The seal assembly is shown in Fig. 24. The seal consists of a solid carbon seal ring with a plain flat face running against spiral grooves in the hard chrome-plated surface of the rotating mating ring. A pressure-balanced split piston ring is used for the secondary seal.

The spiral grooves develop both hydrostatic and hydrodynamic lift to maintain face separation to eliminate rubbing contact. The hydrostatic lift is developed across the seal face when a pressure differential is applied under static conditions. The hydrodynamic lift adds to the hydrostatic lift as rotation starts. The lift force is proportional to the face clearance gap, decreasing as the gap increases. Therefore, the gap is self-adjusting to equalize the lift force and closing force. The theoretical gaps are as follows:

#### Condition

Gap

Static	0.000206 cm (0.000081 in.)
2618 rad/s (25,000 spm)	0.000348 cm (0.000137 in.)
9425 rad/s (90,000 rpm)	0.000508 cm (0.000200 in.)

The theoretical leakage across the seal face is given below:

Speed	Leakage
Static	0.0243 m <sup>3</sup> /minute (0.859 scfm)
2618 (rad/s (25,000 rpm)	0.1124 m <sup>3</sup> /minute (3.97 scfm)
9425 rad/s (90,000 rpm)	0.3310 m <sup>3</sup> /minute (11.69 scfm)

The computer printouts for static, 2618 rad/s (25,000 rpm), and 9425 (rad/s (90,000 rpm) are shown in Fig. 25 through 27.

The piston ring was modified to add a circumferential wave spring around the outside diameter to load the piston ring against the housing sealing surface for improved sealing. The wave spring was 0.0000762 m (0.003 in.) brass shim stock.

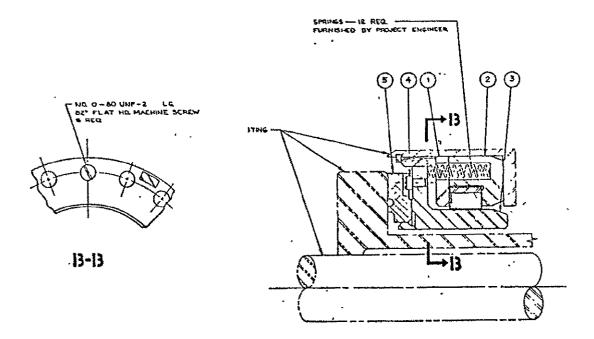


Figure 22. NASA-Designed Oxygen Face Seal With Composite Piston Ring

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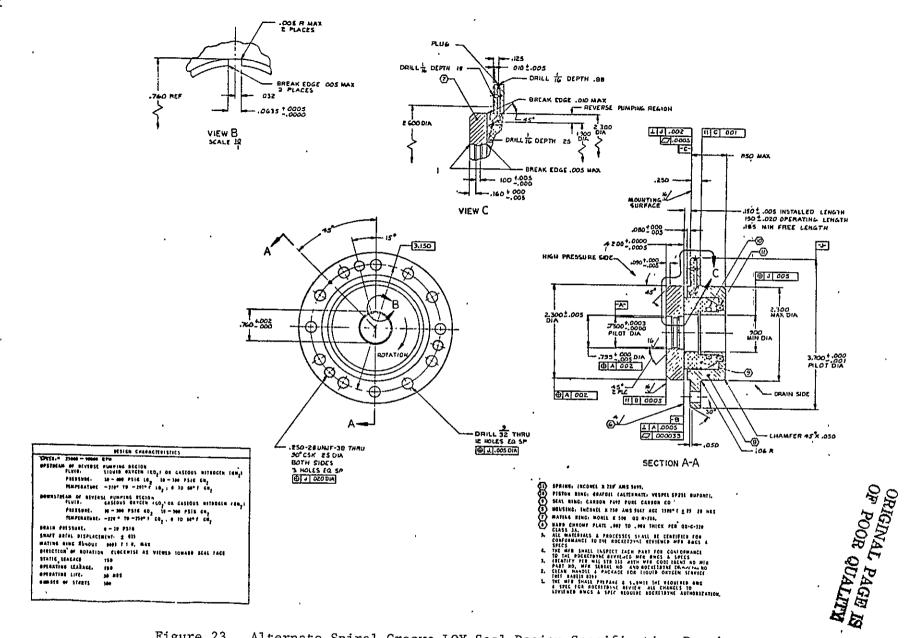


Figure 23. Alternate Spiral Groove LOX Seal Design Specification Drawing

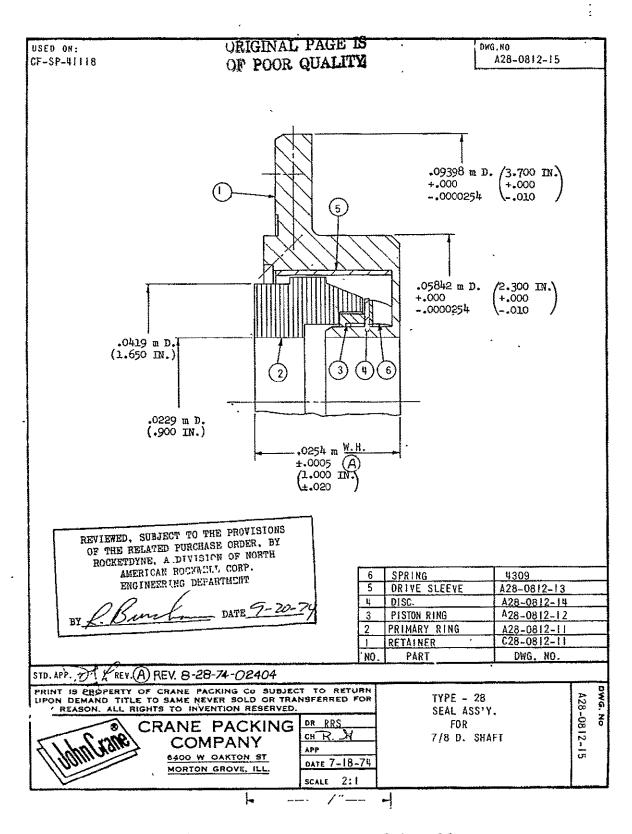


Figure 24. Spiral Groove LOX Seal Assembly

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APOOVE DEPTH	(in)	GPOOVE ANGLE (In)	GPOOVE DIA (1	n) RIPCE WIT	THINCROOVE WINTH	SPRING LOAD	······································
0003500		15.0000	J,0×00	1	. 00000	6.0000	به به المرابع عليه المرابع ال المرابع المرابع
FACE LENGTH	END LE	NGTH O BING PLS	T. SFAL 1.D.	SEAL 0.P.	сит п.п. си	TOFF	*, + wa
.7500	-,4	06015600	, 9000	1.6500	1.4370	.250	
- CLOSE.FORCE	OPEN, FOP	CE YOUNAS MOD.	XRAR YBAR	MOP. OF INERTIA	TOROUE H	P CAP (1n)	* ************************************
177.57	177.	01 3800000,00	.312 .204	.0024187	0. D.	.0000812500	na bland din sjagog – me man a na sjegogijejejeneg j
DT AXIL	DT FACE	INPUT THEP.DEF"	CALTEPP(=)	GRADIENT	THERMAL FY.CO.		1 2
<b>D.</b>	0.	.0	ħ.	0.	.0000033	- ,	۸ گینینیدوینیو ـــــد و مدعو هر چه چه ۲۵۰۵ می ا
THEPITAL DEFL	(rad)'	PPFS.DFFL.(rad)	PRES.MOM. (Inth,	/in) scr#	SCEN		1575257, %)
clear(1)*,000 (5)=.0009803 dear(9)=.000	15070766998 100811999365 108428931314 107690217023	000015094 cface*1,54407720422 cend*.3493033529433 47924 clear(2)*.000 clear(5)*.00097973 2999 clear(10)*.000 DATA CURRENTLY IN	cbot=.49271677393 080902809643772 c 6909015489 clear(3 075957257305502 c	1425 ccbot=.47524 lear(3)=.00008074 7)=.0000727919966 lear(11)=*** clea	731612796 ctall= 4682739819 clear 187993 clear(8)=. xr(19)=*** ;	-6.1570176813593 ;	OF PACE
nod -				Figure 25. S	Static		A B

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Figure 26. 25,000 rpm

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dod=1.65 did=.9 db=1.066 dpr=3. plb=.01 por=120. plr=0. rpm=90000. temp=-297. vis= 1.289800000000E-09 gd=.00035 anr=15.	
dlag#1.03 ratio=1. sload=6. grad=0. thexco*.0000033 flen*.75 elen*406 olen=156 ymod*3800000. sld=.9 odd=1.65 cutod=1.	, 437
cutoff=.25 odla=1.245 len=0.;	
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clearx=.0003	025393351 ;					close=177.5692					·	: • •••
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ח, ה, (ln) t.	. <sup>դ</sup> .(ln)	BAL (In)	PLUG(1n)	PG LIGHT	RANDS	EX.PRES.(pstr)	IN.PRES.(ps)	p) RF	M TEMP(F)	VIS. (1b s	ec/ In sa)	
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FACE LENGTH	END	LENGTH	D RING DIST	. SEA	L 1.D.	SEAL O.D.	CUT O.D.	CUTOFF			•	
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			· •	.332	.204				**********		3 (m. <u>2</u>	
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- THEPHAL DEEL	(rad)	PRESID	FFL.(rad)	PPFS, PD	M.(In16/1	n) SCFM	SCFH			- 42		
.00010036			0015075		126017	11.62970 2093 cctop=8.27	701.1822	hanba D'	200552052677	<	TA DE	
ccback*? 042	50707669	98 cenda.1	493031529433	chots.492	718772012	*5 ccbot≈.47524	731612706 cta	]≈~6,1	570176813593	;	୍ଟିଥି	
c'ear(5)=.00	01992313	9083433 cl	par(5)=.0001	1873869285	171 clear	3)=_00039857299 (7)=_0001979240	9°32]78 clear	(8)=,000	019710950399	235	POOR	
clear(^)*.00 PD YOU ⊎ISH	01962949 To FRFE	0965292 cl THE DATA C	ear(10)=,000) UPPENTLY 18	[954203153 THE PPAGRA	7349 clea 19777 ANS	r(ll)=*** clear VFP MUST RE 11	(12)=*** ; OR 1H*1111		0		05	×
<u>n</u>			••••••				٠		• •	•	PA	**
											AG	

Figure 27. <u>90,000 rpm</u>

## RAYLEIGH PAD FLOATING RING HELIUM SEAL

The design features shrouded Rayleigh step hydrogynamic lift pads on the floatint seal ring inside diameter to center the rings on the rotating mating ring with minimum wear and leakage. The rings are separated by a spring loading the rings against the radial seal faces.

The final design is shown in Fig. 28. The NASA supplied design is shown in Fig. 29. The changes from the NASA design are listed below:

- 1. Use of a bolted, flanged mounting with positive static sealing of the helium purge
- 2. Use of composite metal-banded carbon rings
- 3. Use of two antirotation tangs on the outside diameter of the metal band and slots in the housing
- 4. Use of a wave spring separating carbon rings
- 5. Full pilot under sealing surface of mate sleeve

The materials used in the design are shown in Fig. 30.

The predicted performance at maximum operating conditions of the helium purged shrouded Rayleigh lift pads and seal dam is shown in Fig. 30. The self-acting lift pad characteristics of increased lift and power with reduced film thickness is shown. Leakage also decreases with reduced film thickness.

The thermal analysis results for the helium purge seal components are shown in Fig. 32 and 33. The average temperature at maximum operating conditions was 216.7 K (-70 F.) The results were used in the stress analyses of seal components.

The floating rings are pressure blanaced as close as possible. The forces, pressures, and moments on the floating rings are shown in Fig. 34. The rings are pressure balanced except at the seal dam and seal face. The separating spring force was 3.781 N (90.85 pounds).

The design of the helium-purged seal carbon rings was completed after the diameter characteristics of the rotating mating sleeve as a function of geometry, pilot fits, axial preload, temperature, and speed were evaluated.

Three floating ring configurations were evaluated: solid carbon ring (Fig. 35); composite metal-banded carbon ring (Fig. 36); and segmented or split ring.

The NASA-designed, helium-purged seal, solid carbon ring was compared with a metal-banded carbon composite ring design. Relative radial deflections of the seal rings and the rotating mating sleeve were compared (Table 2 ) to determine the changes in diametral clearances from the installed-to-static chilled conditions and installed-to-maximum operating conditions at 9425 rad/s (90,000 rpm.) The analysis considered the effects of temperature and pressure on the

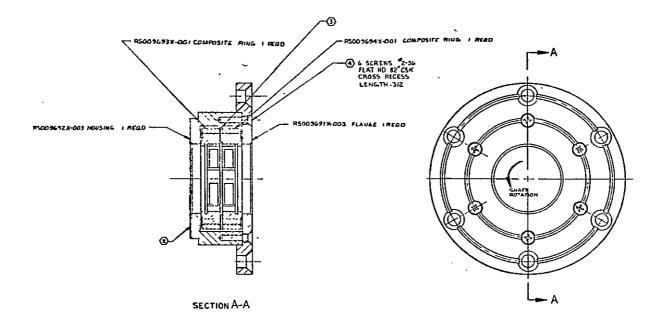


Figure 28. Final Helium Purged Seal Assembly

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Figure 29. NASA Sketch of LOX Face Seal and Helium Purge Seal

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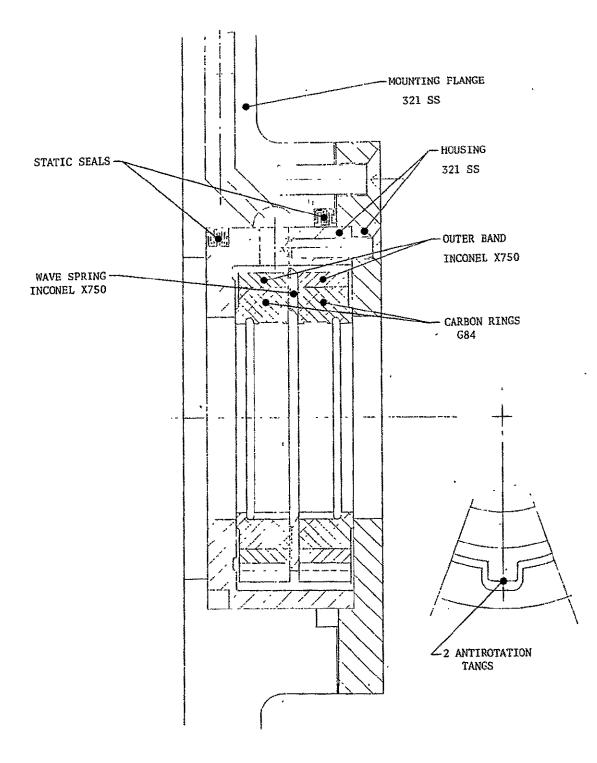


Figure 30. Composite Ring Helium Seal Layout

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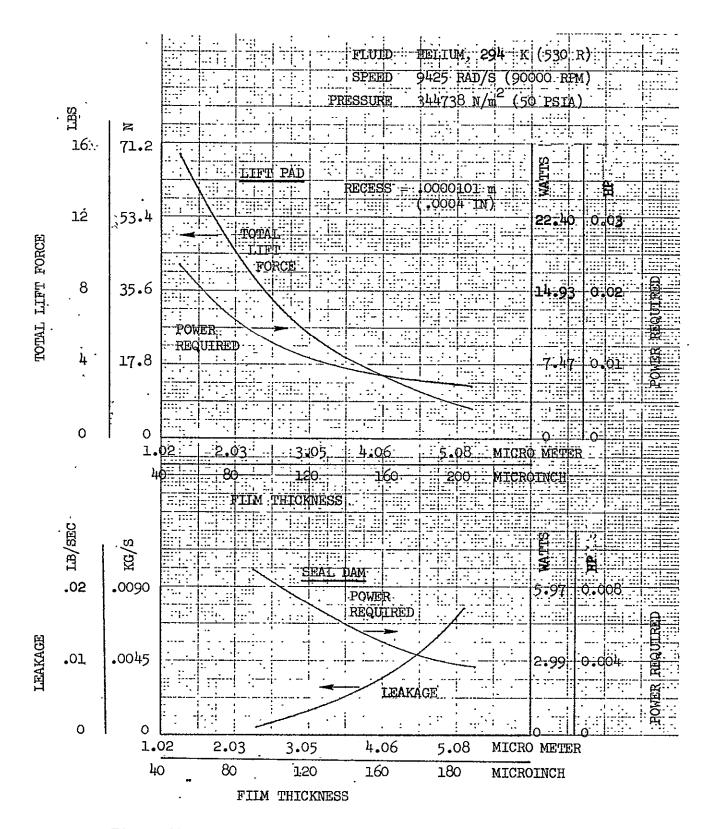
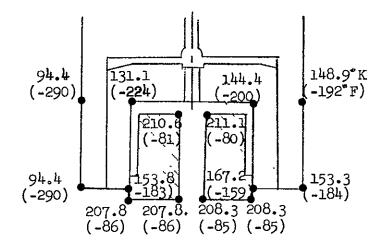


Figure 31. NASA Helium Purge Seal Performance

## AT ZERO RAD/S (ZERO RPM)



AT 9425 RAD/S (90000 RFM)

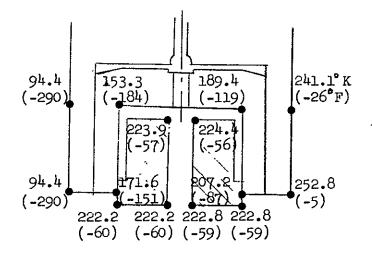


Figure 32. Temperature Profile in Helium Purge Seal Assembly

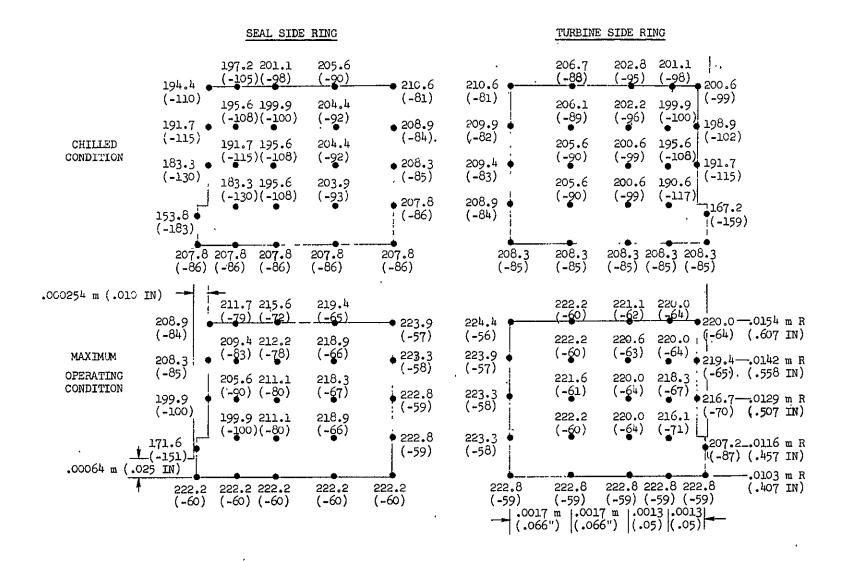


Figure 33. Helium Purged Seal Rings Predicted Temperature Distribution K (F)

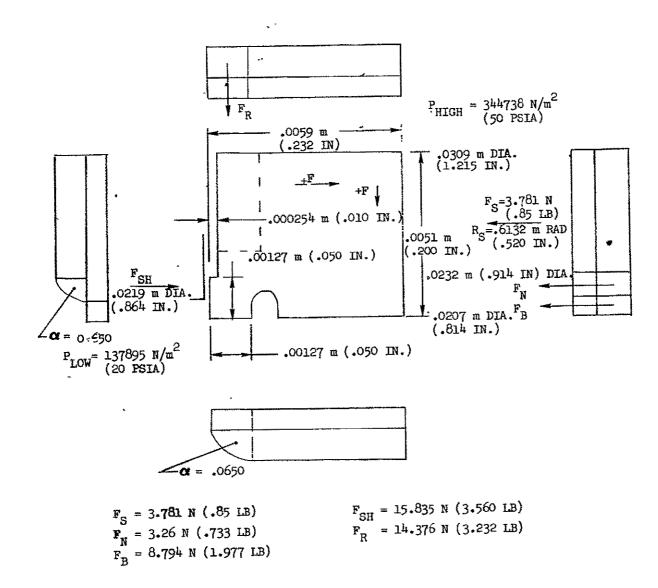


Figure 34. Preliminary Force and Moment Balance Free Body of Carbon Ring - Scale: 10X

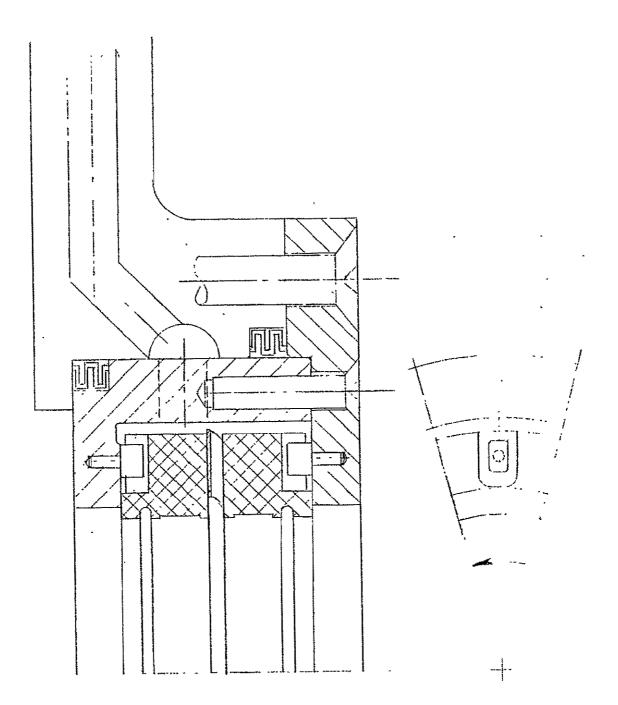


Figure 35. Solid Carbon Ring Helium Seal Layout

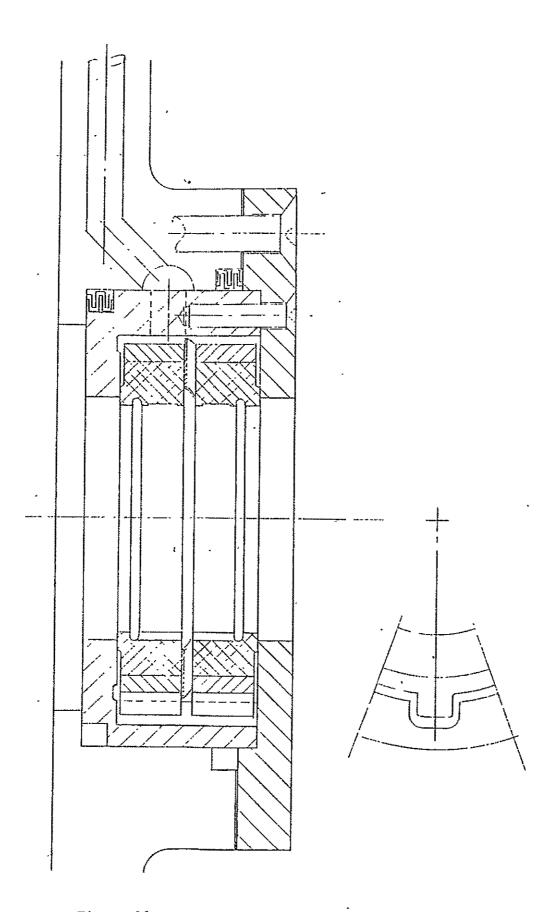


Figure 36. Composite Ring Helium Seal Layout

# TABLE 2. SMALL, HIGH-SPEED SEAL TECHNOLOGY HELIUM-PURGED SEAL RINGS; COMPARISON OF SOLID AND COMPOSITE RINGS

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/Reference:	Ambient Installed Condition
Note:	Dimensions: centimeters
\	(inches)

	Chilled Condition		Maximum Operating Condition	
	Solid	Composite	Solid	Composite
	Ring	Ring	Ring	Ring
LOX Side Seal				
Outer Diameter, Seal Ring ∆Radius	-0.000409	-0.001273	-0.000345	0.001095
	(-0.000161)	(-0.000501)	(-0.000136)	(-0.000431)
Inner Diameter, Shaft Sleeve ∆Radius	-0.002096	-0.002096	-0.000958	0.000958
	(-0.000825)	(-0.000825)	(-0.000377)	(-0.000377)
∆Clearance, radial	-0.001687	-0.000823	0.000612	0.000137
	(+0.000664)	(+0.000324)	(+0.000241)	(-0.000054)
∆Clearance, diametral	-0.003373	+0.001646	0.001224	0.000274
	(+0.001328)	(+0.000648)	(+0.000482)	(-0.000108)
Turbine Side Seal				
Outer Diameter, Seal Ring ∆Radius	-0.000394	-0.001229	-0.000318	-0.001011
	(-0.000155)	(-0.000484)	(-0.000125)	(-0.000398)
Inner Diameter, Shaft Sleeve $\Delta Radius$	-0.002096	-0.002096	-0.000739	-0.000739
	(-0.000825)	(-0.000825)	(-0.000291)	(-0.000291)
∆Clearance, radial	-0.001702	0.000866	-0.000422	-0.000272
	(+0.000670)	(+0.000341)	(+0.000166)	(-0.000107)
∆Clearance, diametral	-0.003404	0.001732	-0.000843	-0,000544
	(+0.001340)	(+0.000682)	(+0.000332)	(-0.000214)

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seal rings and temperature, pressure, rotation, axial bolt load, and pilot press fits on the mating sleeve. The change in diametral clearance for the solid ring was nearly twice that for the composite ring at the chilled conditions. The relative diametral clearance was larger for the solid ring than for the composite ring at the 9425 rad/s (90,000 rpm) maximum operating condition. The composite ring design was selected on the basis of improved clearance control for lower helium leakage and increased hydrodynamic lift potential.

The predicted range of film thickness at maximum operating conditions in the tester is 0.00066 to 0.00173 cm (0.00026 to 0.00068 in.).

The helium seal mating ring was designed to have a constant diameter at maximum operating conditions. Thermal and stress analyses including finite element modules indicated a full pilot was required under the sealing area for dimensional control. Mating sleeve deflections due to rotation, temperature, axial bolt load, and pilot fits are summarized in Table 3. Mating sleeve material is K-monel for oxidation resistance and hard chrome plate for the sealing surface. Provisions were made on the sleeve for Bently depressions, puller holes, and LOX mate ring clamping shoulder.

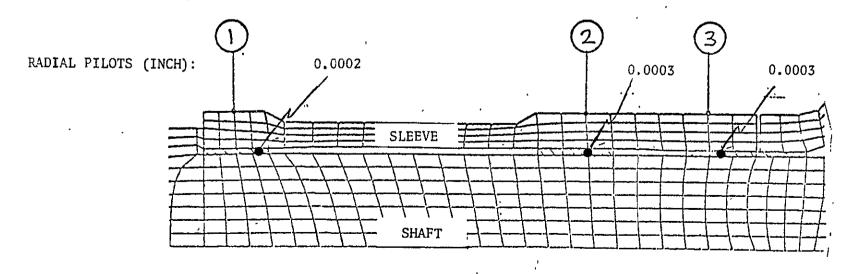
The helium seal housing has a single feed hole for the helium purge flow. Flat, smooth surface finish faces are provided for seal ring faces and the static seal faces. The housing outer inside diameter corners are chamfered to facilitate seal installation. The housing-to-mating ring radial clearance is 0.076 cm (0.030 in.), which would result in a  $7.22/m^3/minute$  (255 scfm) helium flow at 344,738 N/m<sup>2</sup> (50 psi) in case of a catastrophic seal failure.

## TABLE 3. HELIUM PURGED SEAL ROTATING MATING SLEEVE RADIAL DISPLACEMENTS SUMMARY

Location Condition	At Mating 1 Ring Shoulder	At Seal Side Ring	At Turbine 3 Side Ring
Sleeve *	(Inch) +.000148	(Inch) +.000218	(Inch) +.000207
Sleeve + Assembly **	+.000168	+.000221	+.000213
Chilldown	000658	000604	000612
Temperature + Speed	000585	000156	000078
		!	

\* Values shown are installed with radial interference, pre-installation values are all equal to 0.00.

\*\* Assembly indicates a preload of 3500 lbs.



## TESTER DESIGN

The seal tester assembly is shown in Fig. 37. The tester is a modification of an existing LOX seal tester designed to simulate, as closely as possible, the operating conditions in a liquid oxygen turbopump including an acceleration rate of 4189 rad/s/s (40,000 rpm/sec) with the liquid oxygen pressure increase proportional to the square of the speed. A braking system is also provided to decelerate the tester from full speed to zero rpm in 4 seconds. The tester consists of a simulated turbopump overhung rotating shaft, mounted in two preloaded angular contact bearings. The tester bearing size was minimized consistent with the load, life, stiffness, and critical speed requirements to provide the minimum DN value for safe operation in liquid oxygen. The DN value for the selected 15-mm bearings at the 9425 rad/s (90,000 rpm) operating point is 1.35 million. The bearing design is shown in Fig. 38, and predicted bearing B<sub>1</sub> life is shown in Fig. 39. The bearings are lubricated and cooled by liquid oxygen which flows inward through each bearing with a drain located in the cavity between the bearings. LOX flow to each bearing is 0.0189 m<sup>3</sup>/minute (5 gpm) and is entirely separate from the flow to the test seals.

The tester shaft is accelerated and driven at the design speed of 9425 rad/s (90,000 rpm) by a 0.064 m (2-1/2 in.) diameter radial inflow turbine mounted on the overhung end of the shaft. Drive gas for the turbine is ambient nitrogen which is supplied to the rotor by four 0.0064 m (0.25 in.) diameter forward and two 0.0064 m (0.25 in.) reversing nozzles. Performance characteristics of the turbine are shown in Fig. 40. A close clearance Kel-F labyrinth seal was incorporated at the turbine rotor OD to lower the pressure in the adjacent seal drain area. The test seals (two) are located on the overhung portion of the shaft between the turbine and the bearing to stimulate, as close as possible, the actual turbopump installation.

An'additional labyrinth seal is located between the test seals and the outward turbine bearing and serves two purposes:

- 1. It separates the upstream seal cavity from the bearing cavity.
- 2. The diameter is sized to help balance the tester thrust loads.

Drains are provided in the tester housing in the cavities on either side of the purged helium seal. Helium purge gas fed to the helium seal is allowed to leak to both cavities where, on one side, it sweeps any simulated turbine leakage gases out through a drain port, and on the opposite side, it sweeps any leakage from the LOX fluid film face seal out through a separate drain port.

Critical speed analysis was performed on several reiterations of tester shaft designs to provide safe operation at the required test speeds. The final design has a weight added between the bearings to shift the second critical speed further from the nominal operating speed. The first critical 1780 rad/s (17,000 rpm) is below the minimum 2618 rad/s (25000 rpm) test speed, and the third critical 5136 rad/s (115,000 rpm) is above the maximum 9425 rad/s (90,000 rpm) test speed. Other test speed was selected at 6279 rad/s (60,000 rpm) to maintain adequate margin once the 4919 rad/s (47,000 rpm) second critical

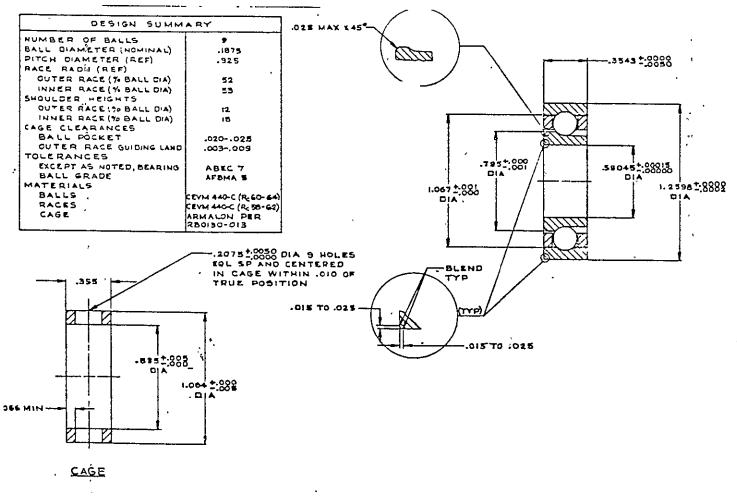


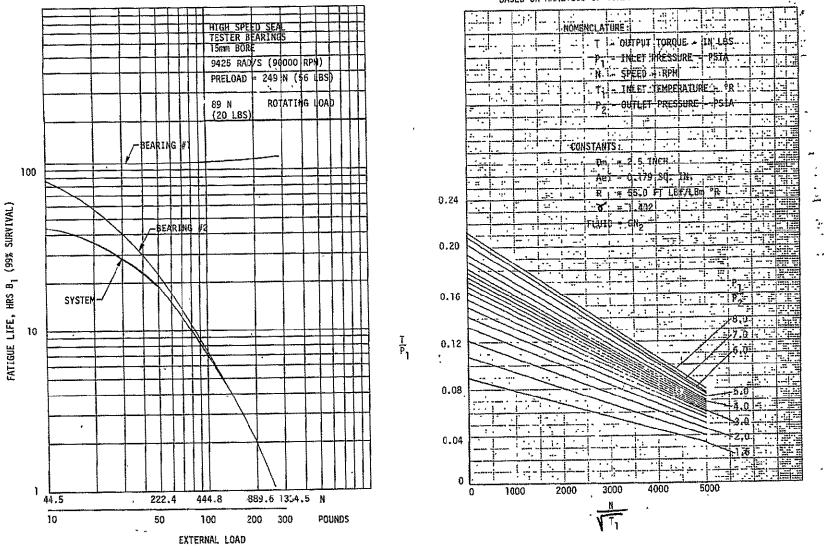
Figure 38. Tester Bearing

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## BASED ON ANALYSIS OF TURBINE CALIBRATION TESTS



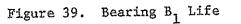


Figure 40. Mark 36 Turbine Performance Map

speed. Results of the critical speed analysis are shown in Fig. 41 where critical speeds are plotted against bearing stiffness.

## THERMAL ANALYSIS

A thermal analysis was made of the seal tester for chilled zero speed conditions and also the design operating conditions at 9425 rad/s (90,000 rpm). The analysis indicated the bearing and LOX seal cavities were maintained at LOX temperature 90 K (-296 F) during operation. The LOX side of the helium seal housing also remained at essentially LOX temperature at operating conditions. The helium mating ring sleeve temperature increased from 201 K (-96 F) at chill to approximately 222 K (-60 F) at the 9425 rad/s (90,000 rpm) operating condition. The turbine rotor also increased in temperature from 197 K (-105 F) at chill to 284.8 K (+53 F) at operating conditions.

### THRUST BALANCE

To achieve the required minimum 10-hour life of the tester bearings, the thrust load on the bearings had to be maintained at a maximum of approximately 334 N (75 pounds) a total shaft thrust load at 9425 rad/s (90,000 rpm) was achieved by pressure balancing the system. Bladed housings were incorporated on either side of the shaft weight to balance pressure across the rotating element, and the diameter of the labyrinth between the bearing and the test seal was sized to achieve a total thrust load of 334 N (75 pounds) toward the turbine. Fig. 42 is a summary of the calculated thrust loads in the tester.

#### MATERIALS

The materials selected for the tester are shown in Fig. 43. The stainless and nickel steels were selected because of their LOX compatibility, resistance to oxidation and their strength, ductibility, and hardness in the required operating environment. The 321 stainless steel used in the bearing housing, bearing carriers, and helium seal housing exhibits high ductility and adequate strength at cryogenic temperatures. Inconnel 718 was used for the turbine rotor, shaft, shaft weight, LOX seal mounting ring and LOX seal housing. The K-monel used in the LOX seal mating ring and the helium seal mating sleeve has high resistance to oxidation and can be chrome plated.

A286 stainless steel is used for the bearing end shaft nut and seal end bearing carrier nut. The 6061 aluminum drive turbine manifold was an existing part and required special Invar washers for mounting to provide gasket sealing with the high thermal contraction of aluminum.

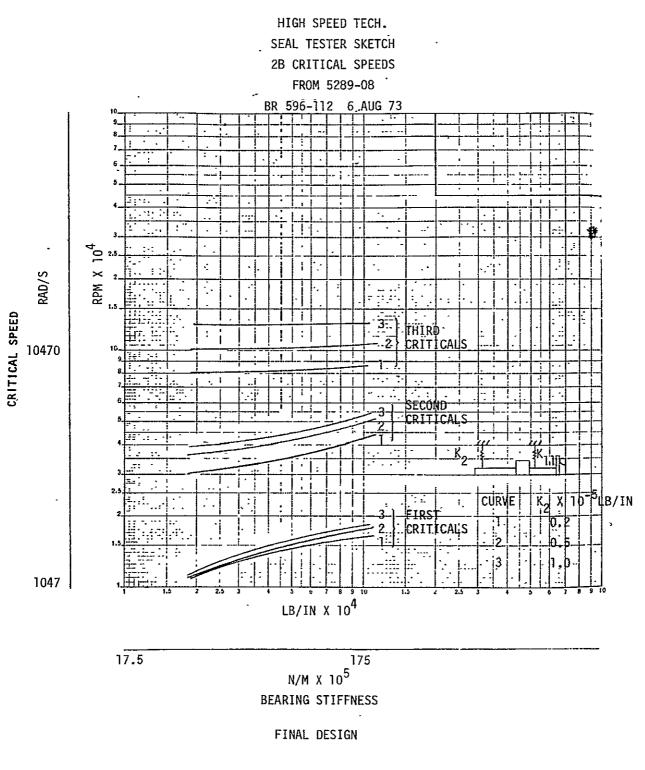
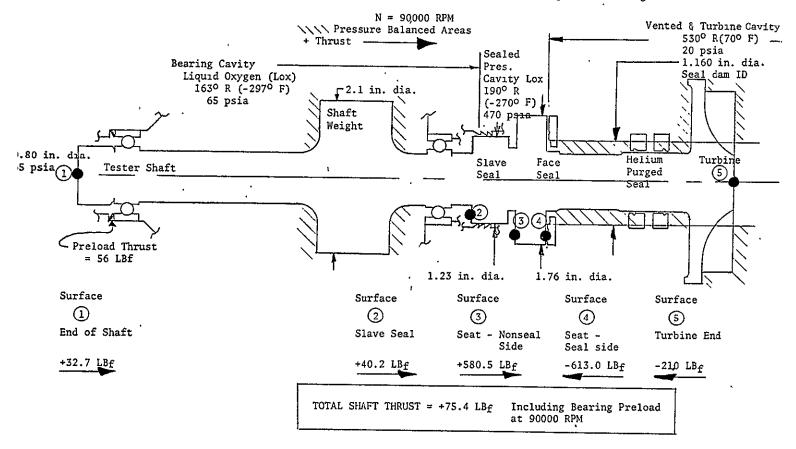
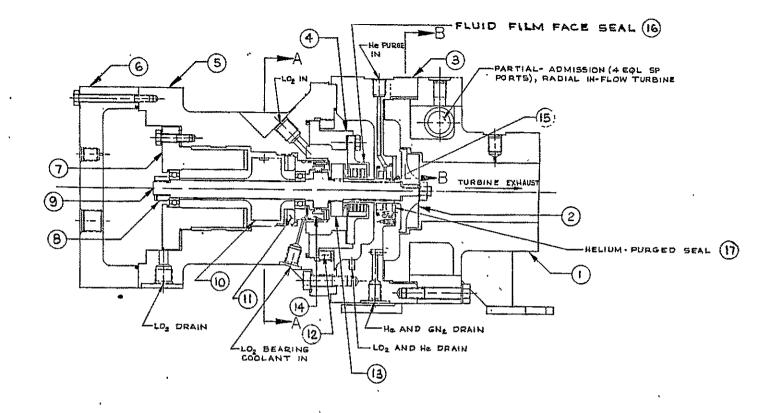


Figure 41. Critical Speeds



## Configuration = Smooth disk, bladed housing at shaft weight

Figure 42. Small, High-Speed Seal Tester Shaft Axial Thrust Summary



LEGEND	MATERIAL
1	6061 ALUM
2	INCO 718
3	321 CRES
	INCO 7 18
5	321 CRES
6	321 CRES
7	321 CRES
8	A286 CRES
9	INCO 718 .
10	INCO 718
11	A286 CRES
12	A286 CRES
13	K-MONEL
14	321 CRES

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LEGEND	MATERIAL
15	K-MONEL
16	INCO 718
17	B2I CRES

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Figure 43. Small High Speed Seal Tester Materials

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FOLDOUT FRAME

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FOLDOUT FRAME

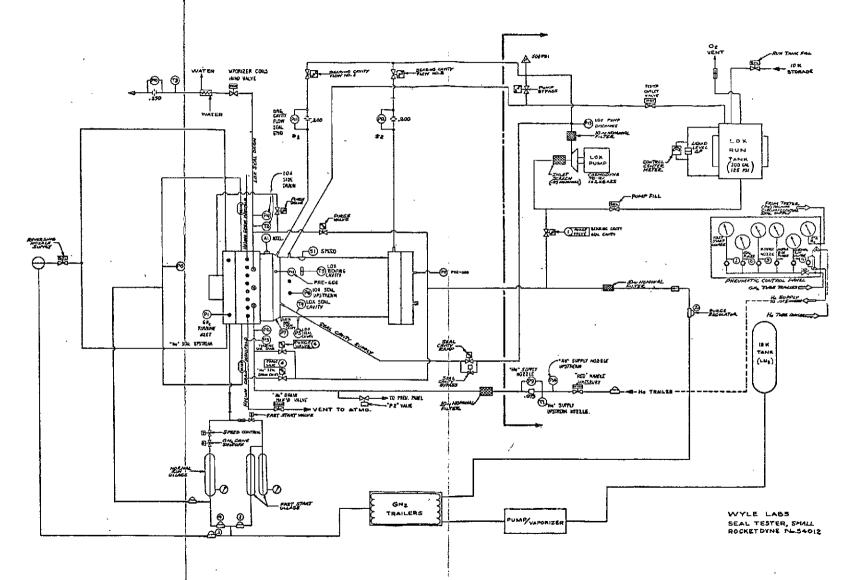


Figure 44. Seal Tester Flow Schematic

#### TEST FACILITY

#### DESCRIPTION

Seal testing was accomplished at Wyle Laboratories, Norco, California. Capa-541 titles include 106 m<sup>3</sup> (28,000 gallons) and 49.2 m<sup>3</sup> (13,000 gallons) vacuumjacketed LH<sub>2</sub> and LOX storage with steady-state flow to 41.63 m<sup>3</sup>/minute (11,000 gpm); 49.2 m<sup>3</sup> (13,000 gallons) and 37.85 m<sup>3</sup> (10,000 gallons) vacuum storage tanks for LOX and LM<sub>2</sub> with ready-state flows to 37.85 m<sup>3</sup>/minute (10,000 gpm); and high-pressure pneumatic systems with air/aitrogen flowrates to 45.35 kg/s (100 lb/sec), and helium flowrates to 6.8 kg/s (15 lb/sec). A schematic of the seal test setup is shown in Fig. 44.

Liquid oxygen was supplied to the tester from a  $1.14 \text{ m}^3$  (300 gallons) tank by a Cosmodyne Model TC-21 pump through a 10-micron filter. The flow was split into bearing coolant flow and LOX seal flow.

Flow to each of the two bearings was measured by calibrated orfices and controlled by motorized values. LOX seal pressurization was also controlled by an upstream motorized value. For the ramp test, seal cavity pressure rise from  $88.96 \text{ N/m}^2\text{g}$  (20 psig) to  $178 \text{ N/m}^2\text{g}$  (400 psig) at a rate proportional to the square of the rotational speed, a bypass in conjunction with a solenoid value was used. A preramp pressure was established with the bypass value; the solenoid was then actuated to supply the  $1780 \text{ N/m}^2\text{g}$  (400 psig) at the required rate. Helium purge flow was supplied from a pressurized tank and regulated to provide  $133.45 \text{ N/m}^2$ a (30 psia) at the tester.

Gaseous nitrogen for the tester drive turbine was supplied from a pressurized tank. Solenoid and motorized valves were used to control the  $GN_2$  pressure to achieve the desired test speed. For the fast-start tests, an acceleration rate of 4189 rad/s/s (40,000 rpm/sec) additional  $GN_2$  tank was plumbed into the drive system. The additional  $GN_2$  supply was cut in at turbine start and automatically deactivated when the tester operating speed was reached. Fast-start rane rate was controlled by varying the fast-start tank pressure.

The requirement to decelerate the tester from operating speed to zero speed in 4 seconds was achieved by simultaneously closing the LOX seal supply and turbine solenoid valves and actuating the solenoid valve to supply GN<sub>2</sub> to the two turbine reversing nozzles for 2 seconds, and then the tester brake actuators for 1.5 seconds. The valve opening times were controlled by a cam finer activated at shutdown. The turbine reversing nozzles and brake actuators were each supplied by separate GN<sub>2</sub> tanks and cank pressures were varied to set deceleration rates. During the decleration time and the use of the brake system was discontinued.

Gaseous nitrogen purges were used both before and after each test to prevent moisture contamination of the tester. The purges were also activated between tests whenever a significant delay resulted in LOX pump shutdown. Purge gas was supplied to the bearing cavity, LOX seel upstream and downstream cavities, and the helium seal upstream and downstream cavities. The tester installed in the facility is shown in Fig. 45 through 48. Figure 45 is a photograph of the facility showing the overall layout of the test site and the relationship of the tester to the facility main fluid supply systems. Figure 46 is a view of the tester looking into the turbine exhaust and shows the location of the main flow control valve. Figure 47 is a closeup view of the tester showing the LOX seal supply line and the bearing drain line. Figure 48 illustrates the location of the bank of pressure transducers and also shows the location of the orifice used to measure LOX seal drain cavity leakage. Figure 49 is a photograph of the control panel and the strip charts used to record some of the data.

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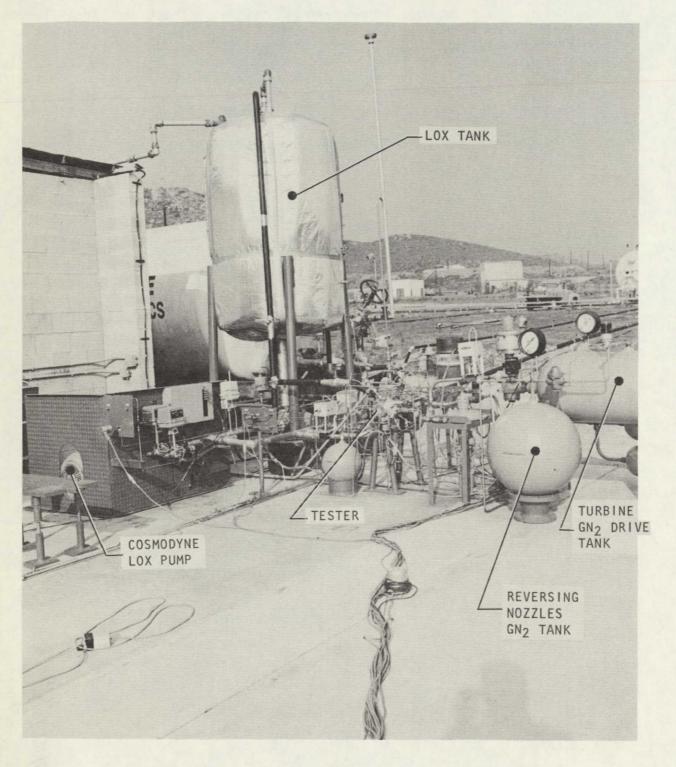
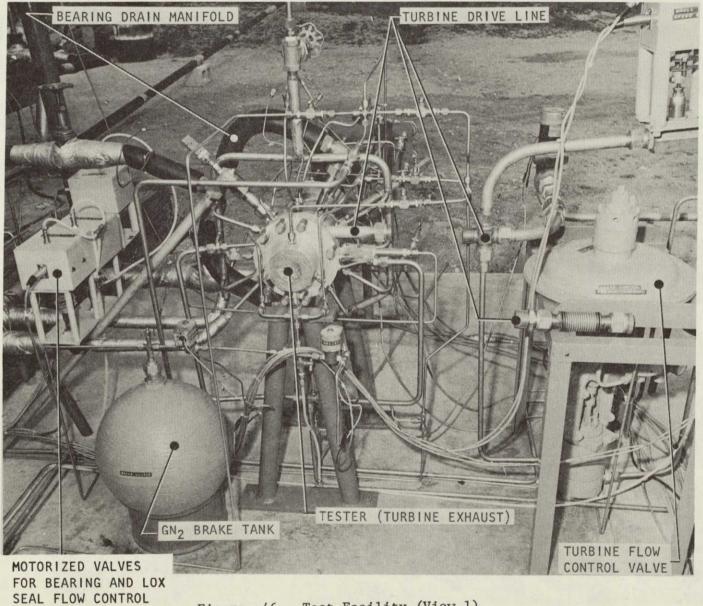


Figure 45. Test Facility (overall)



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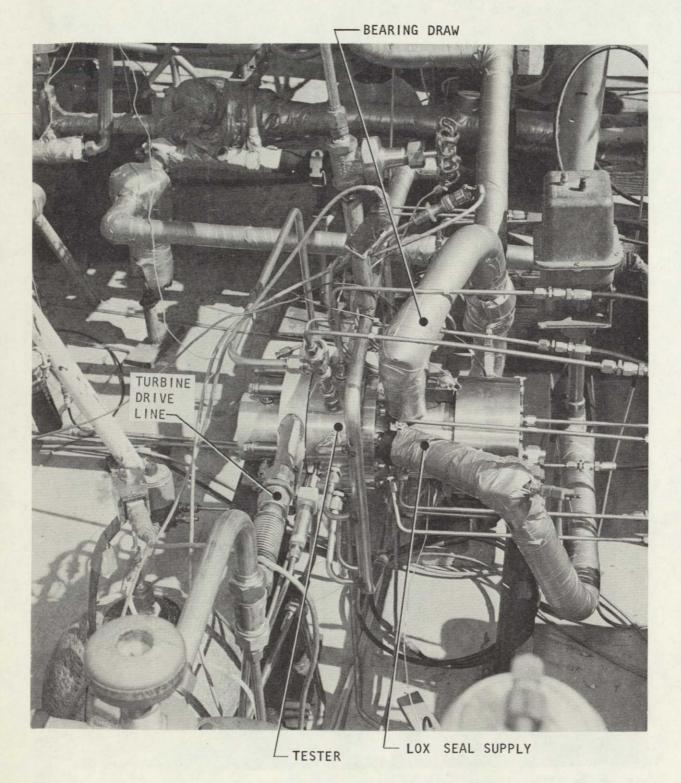


Figure 47. Test Facility (view 2)

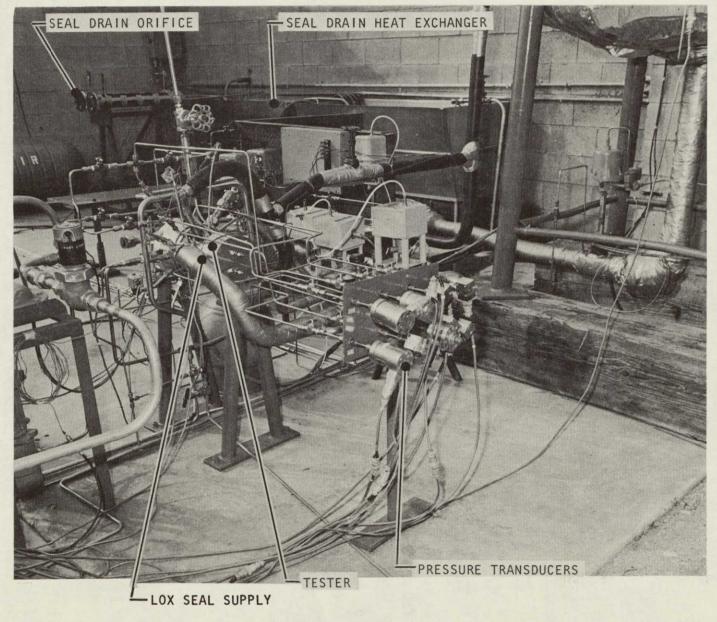


Figure 48. Test Facility (View 3)

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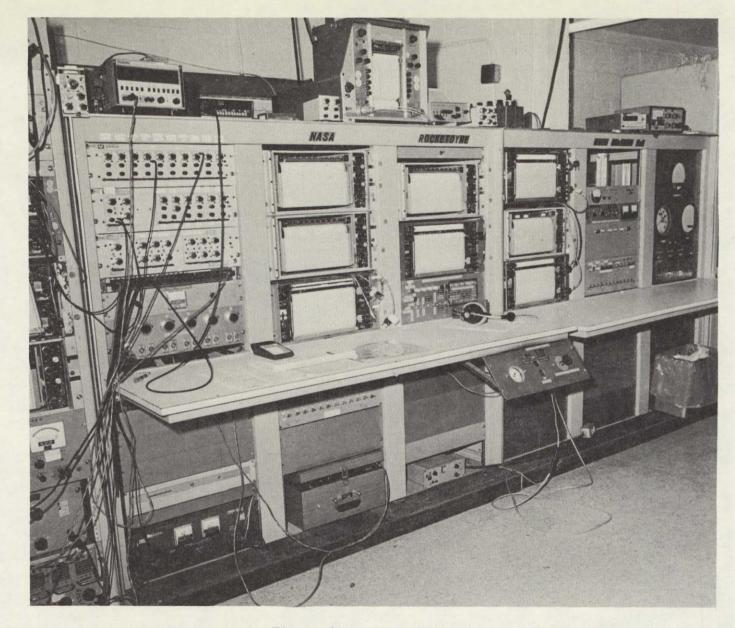


Figure 49. Control Panel

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#### TEST HARDWARE

#### TESTER HARDWARE

The seal tester (Fig. 50), part number RS009661X, was assembled by Rocketdyne and supplied to the test site ready for installation. The tester assembly was driven by a built-in  $GN_2$  turbine. Instrumentation provided with the tester included a speed pickup which senses the passing of three holes per shaft revolution, a LO<sub>2</sub> temperature bulb to measure bearing cavity temperatures, and two Bently probes to measure shaft deflection.

The tester hardware is shown in Fig. 50. Components shown in the photograph include: (1) the shaft assembly, (2) the slotted cover end bearing carrier, (3) the bearing housing, (4) LOX seal assembly, (5) helium seal housing and seal assembly, and (6) the turbine manifold.

#### SEAL HARDWARE

The machined metal bellows LOX face seal assembly is shown in Fig. 51. The helium purged seal assembly is shown in Fig. 52.

Figure 53 is a photograph of the piston ring LOX face seal assembly. The spiral groove LOX face seal assembly is shown in Fig. 54.

#### TEST REQUIREMENTS

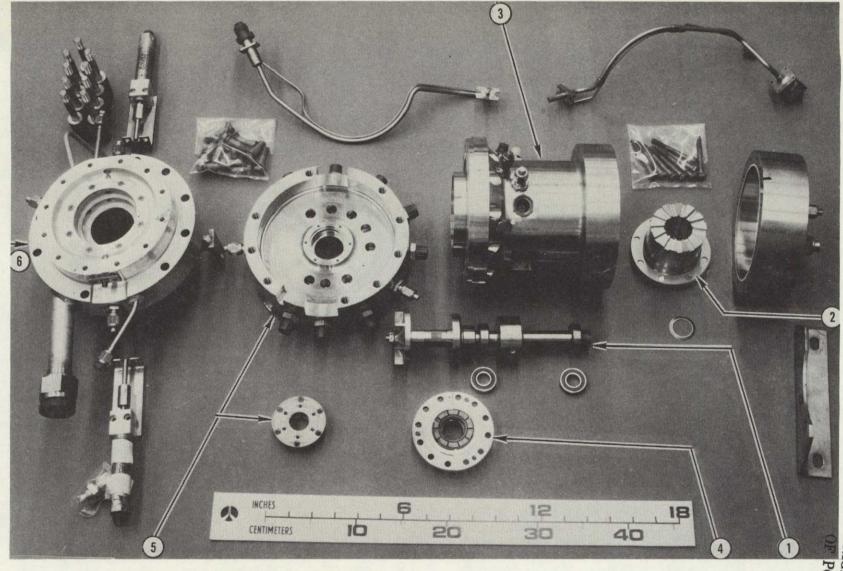
#### INSTRUMENTATION

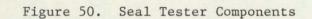
Instrumentation requirements including the redline limits are listed in Table 4. Data were recorded continuously on direct inking graphic recorder charts and/or FM tape. Location of instrumentation taps on tester is shown in Fig. 55.

The seal leakage measurements were recorded with calibrated orifice  $\Delta P$  measurements. The LO<sub>2</sub> seal leakage was drained through a heat exchanger to vaporize the fluid and raise the temperature to approximately atmospheric prior to the flow measurement.

The helium purge flow was measured continuously upstream of the circumferential seal. The helium leakage into the  $LO_2$  side seal drain was assumed to be half of the total purge flow in.

Prior to conducting the two test schedules, two operational checkout tests of 6 minutes duration each were conducted to measure shaft deflections using the Bently transducers (2 each) in place of the helium seal. The testing included a slow acceleration test of 4189 rad/s/minute (40,000 rpm/minute) and a fast acceleration test of 4189 rad/s/s (40,000 rpm/sec) from 0 to 9425 rad/s (90,000 rpm). Data obtained were analyzed to ensure that shaft deflections are within the operating limits prior to commencing the test schedule I preliminary check-out tests. The speed, accelerometer, and Bently data were recorded on FM tape. All other data were recorded on graphic recorder charts.





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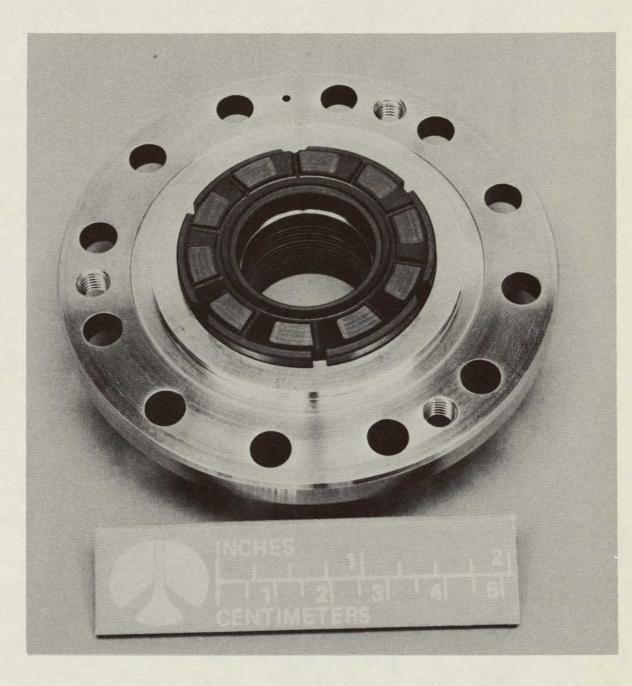
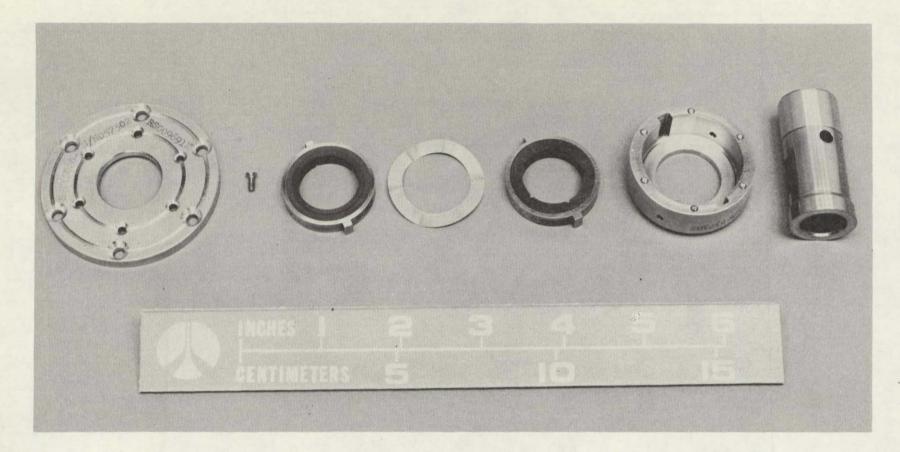


Figure 51. Machined Metal Bellows Lox Face Seal Assembly

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Figure 52. Helium Purged Seal Assembly

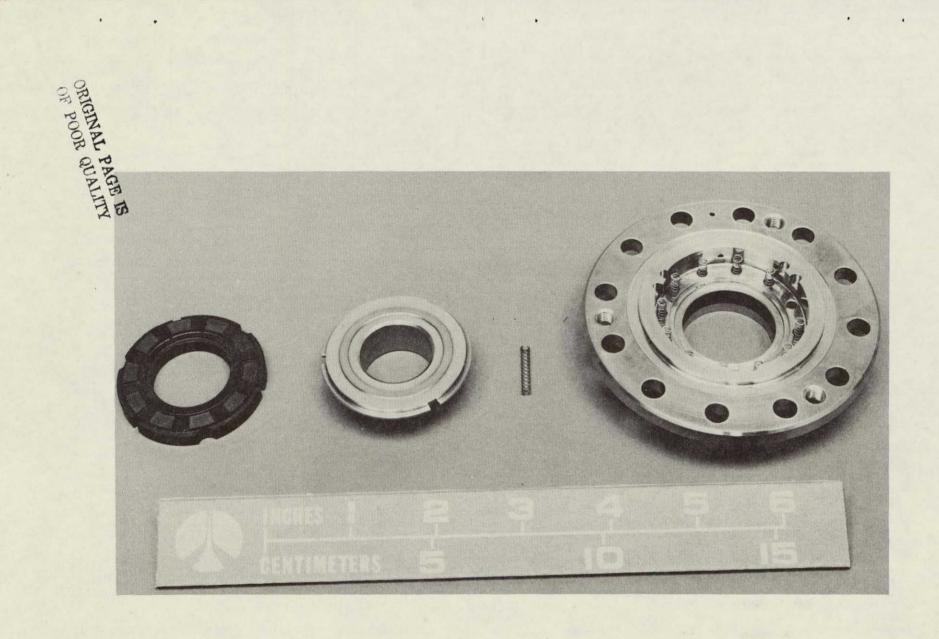


Figure 53. Piston Ring LOX Face Seal Assembly

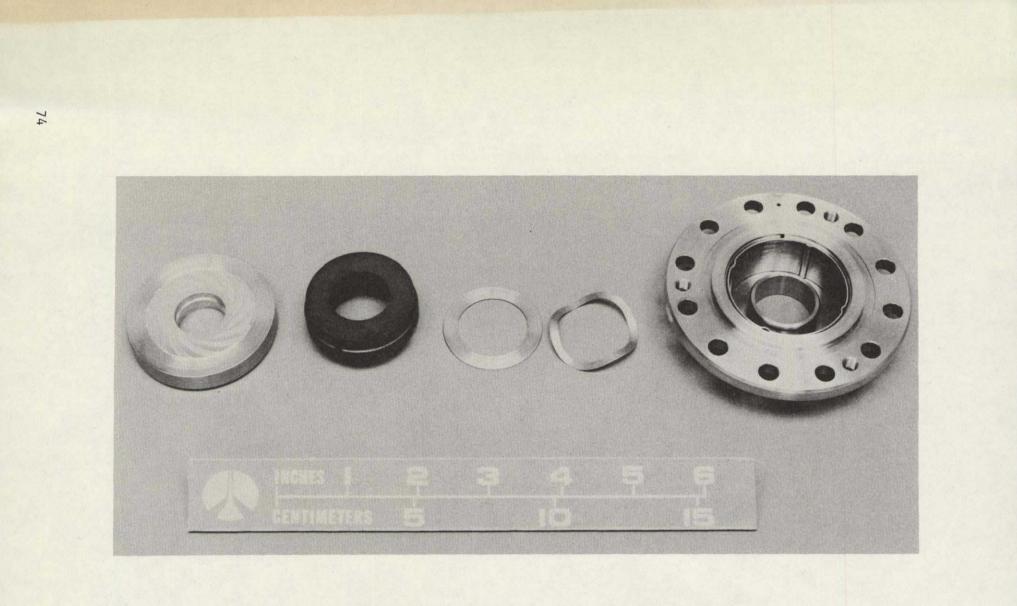


Figure 54. Spiral Groove LOX Face Seal Assembly

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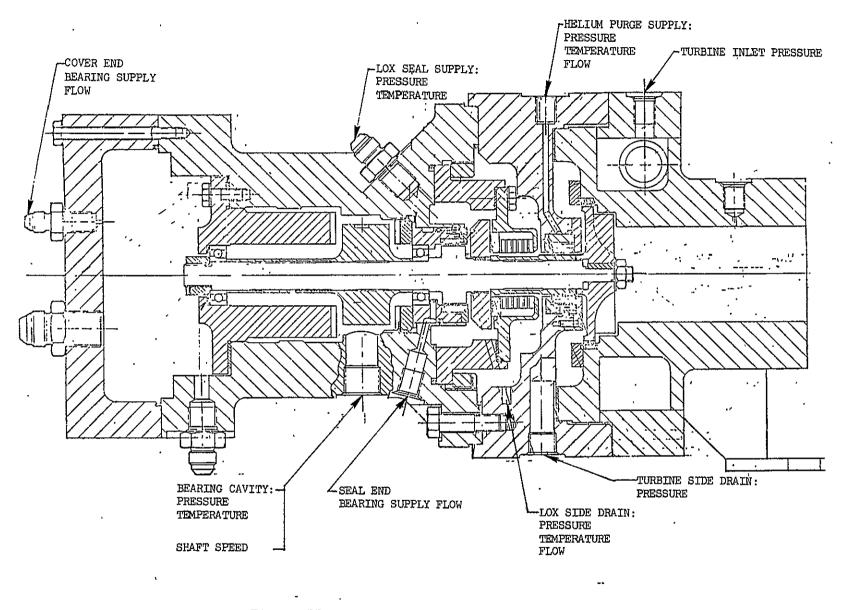


Figure 55. Instrumentation Tap Location

#### TEST SCHEDULE I PRELIMINARY CHECKOUT

Preliminary checkout tests were conducted on each seal assembly using clean, dry, gaseous nitrogen at room temperature at the test conditions shown in Table 5. The LOX seal GN<sub>2</sub> pressure and the helium seal purge pressure were applied prior to start of rotation.

Test Point Number	Shaft Speed, rad/s (rpm)	Face Seal Pressure, ` N/m <sup>2</sup> d (psia	Circumferential Seal Pressure, N/cm <sup>2</sup> d (psia)	Time minutes
1	2618 <u>+</u> 524 (25,000 <u>+</u> 500)	275,790 <u>+</u> 34.474 (40 <u>+</u> 5)	206,843 <u>+</u> 34.474 (30 <u>+</u> 5)	6
2	6282 (60,000)	1.034,214 <u>+</u> 103,421 (150 <u>+</u> 15	206,843 (30)	
3	6282 (60,000)	1,723,689 (250)		
4	7329 (70,000)			
5	8376 (80,000)		-	
6	9425 (90,000)	ł		
7	8376 (80,000)	2,068,429 (300)		
8	9425 (90,000)	1,378,951 (200)		
9		2,068,429 (300)		
10			68,948 (10)	
11	ł	<b>F</b>	137,895 (20)	
12	9425 (90,000 +500)	2,068,427 <u>+</u> 10,321 (300 +15	206,843 +34,474 (30 <u>+</u> 5)	6

TEST	5.	TEST	CONDITIONS
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## TEST SCHEDULE II LO2 TESTING

 $LO_2$  tests were conducted on each seal assembly using  $LO_2$  as the sealed fluid. Each of the seal designs was tested at the following test conditions:

Test Point Number	Shaft Speed, rad/s (rpm)	LOX Seal Pressure, N/m <sup>2</sup> d ' (psid)	Helium Seal Pressure, N/m <sup>2</sup> d (psid)	Time, minutes
Yan	0 to 8062 +105 (0 to 77,000 <u>+</u> 1000)	137,895 to 2,757,903 <u>+</u> 103,421	206,843 <u>+</u> 34,474 (30 <u>+</u> 5)	3
	t point 1 300 times ch 5 hours, or as rec			

The LOX face seal pressure values above are during LO<sub>2</sub> seal operation at the maximum shaft operating speed. During acceleration, the LOX face seal pressure was obtained by ramping the LO<sub>2</sub> seal cavity pressure as a function of the shaft speed squared. During deceleration, the LOX face seal pressure was vented. The helium circumferential seal pressure will be applied prior to start of rotation and will remain on until after rotation stops.

The turbine forward, reverse, and brake actuator valves were adjusted to permit an acceleration rate of 4188 rad/s (40,000 rpm/sec) and deceleration to 0 rpm within 4 seconds as shown in Fig. 56. The forward turbine valve was opened to supply drive GN<sub>2</sub> to the turbine at the ramp rate of 4188 rad/s (40,000 rpm/sec). After operating the tester at 8062 rad/s (77,000 rpm) for 3 minutes, the forward valve closes and the reversing valve opens which supplies GN<sub>2</sub> to the turbine in the reverse direction. When the speed has been reduced to about 1047 rad/s (10,000 rpm), in about 2 seconds, the reversing valve closes and the brake actuator valve opens. The brake actuator pressure was adjusted to provide stopping of the shaft rotation in about 1.5 seconds.

#### PRETEST PROCEDURES

Prior to the start of the preliminary checkout and liquid oxygen testing, and after each seal modification or rework, the following information was obtained on each seal assembly to be tested:

 Static seal leakage at differential pressures across the face seal of 172,369, 344,738, 689,476, 1,378,951, and 2,068,427 N/m<sup>2</sup>/d (25, 50, 100, 200, and 300 psid) in the test rig.

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- 2. Dam heights, pad depths, and/or other pertinent dimensions measured by surface profile traces.
- 3. Surface profile traces and optical readings of flatness of seal mating surfaces.
- 4. Torque required to overcome the static friction of the seal assembly to initiate rotational motion of the shaft.

#### POSTTEST PROCEDURES

All items in the pretest procedure, as described above, were determined at the completion of each seal assembly testing for each test schedule. A careful visual examination was made of each seal (e.g., for cracks, excessive wear, distortions). Photographs of normal and unusual posttest conditions were taken.

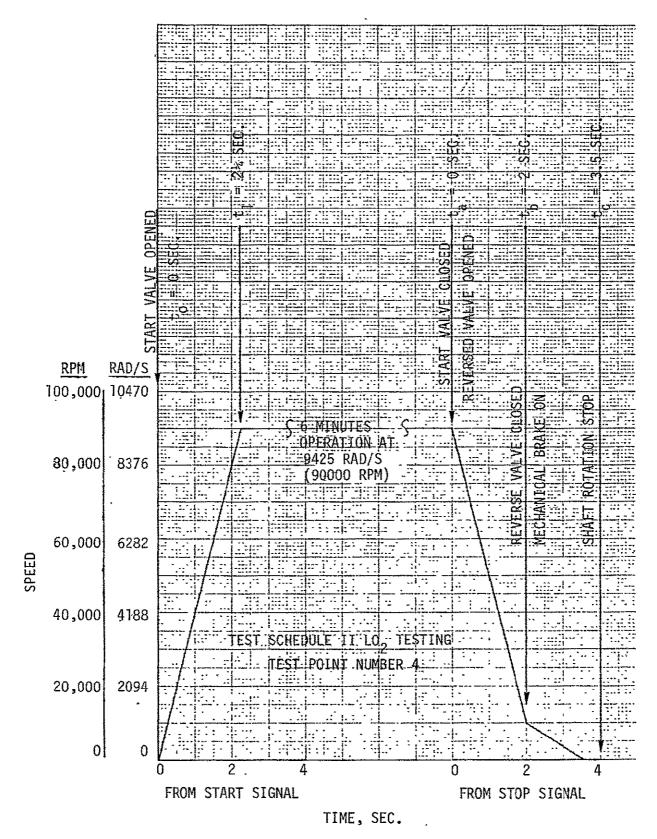


Figure 56. Typical Seal Acceleration and Deceleration Ramp Rates

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TEST SUMMARY

The test summary is given in Table 6.

HARDWARE AND INSPECTION SUMMARY

The LOX seal hardware summary is given in Table 7. The helium seal hardware summary is given in Table 8. The LOX seal inspection summary is given in Table 9 for English units and Table 10 for SI units. The helium seal inspection summary is given in Table 11 for English units and Table 12 for SI units.

DATA SUMMARY

The test data summary is given in Table 13 for English units and Table 14 for SI units.

### TABLE 6. SMALL HIGH SPEED SEAL TEST SUMMARY

Build	Tests	Starts	Time, Hinutes	Objective	· LOX Seal	Hellum Seal	Remarks
1	, 035-045	11	2.7	GN <sub>9</sub> Checkout 2618 rad/s (25,000 rpm), 275,790 N/m <sup>2</sup> (40 ps1) at LOX seal	Rayleigh, bellows, new, S/N Ol	Floating ring, new, S/N 01	Completed on test at 2618 rad/s (25,000 rpm), 461,949 $N/m^2$ , (67 psi) LOX seal pressure, 6 minutes, tried for 6282 rad/s (60,000 rpm), 1,034,214 $N/m^2$ (150 psi) LOX seal pressure when excessive leakage occurred
2	045-054	9	10,6	GN <sub>2</sub> checkout	Rayleigh, piston - ring, new \$/N Oi	Floating ring, no rework, S/N Ol	Completed 6-minute test at 9425 rad/s (90,000 rpm), 1,075,582 N/m² (156 psig) LOX seal pressure; found bearing failure at disassembly; seals good condition
3	055-077	23	73.8	GN <sub>2</sub> checkout	Rayleigh, piston ring, S/N Ol new carbon and mate	floaring ring, no rework, S/N 01	New bearing preicad spring design, com <sup>.</sup> pleted GN <sub>2</sub> checkout on Rayleigh LOX seal
4	078-088	1)	20,4	GN <sub>2</sub> checkout	Spiral groove, new, S/N Ol	Floating ring, no rework, S/N Ol	Scheduled Inspection, seals good condition
5	089-092	4	13.5	GN <sub>2</sub> checkoùt	Spiral groove, no rework, S/N Ol	Floating ring, no rework, S/N 01	Bearing failure damaged seals; seal per- formance good before failure
6	093-111	19	66.4	GN <sub>2</sub> checkout	Spiral groove, new, S/N 02	Floating ring, new, S/N 04	Test speed reduced to 8062 rad/s (77,000 rpm); used 0.000013 m (0.0005 in.) smaller bearing balls; completed GN <sub>2</sub> checkout on spiral groove LOX seal; seals and tester good condition
7	112-186	76	84,4	LOX checkout	Spiral groove, carbon and mate lapped flat, S/N O2	Floating ring, new, S/N 03	Posttest dry spin failed bearings and damaged scals, seal performance good before failure
8	187-332	146	268.4	LOX accelerations tests	Spiral groove, new	Floating ring, new, S/N 02	Bearing failure, seals damaged, seal per- formance good before failure
9	333-412	. <sup>80</sup>	150.0	LOX acceleration tests	Rayleigh, piston ring, new, S/N O2	Floating ring, from build 6, no rework S/N 04	Scheduled inspection, excellent condition, hellum seal pads worn away
10	413-517	105	157.7	LOX acceleration tests	Rayleigh, piston ring, no rework, S/N 02	Floating ring, no rework, S/N 04	Scheduled Inspection, LOX seal in excellent condition
n	518-559	42	72.9	LOX acceleration tests	Rayleigh, plston ring, no rework,	Floating ring, no rework, S/N 04	Inspection due to high vibration level, LOX seal in excellent condition
12	560-632	72	155.1	LOX acceleration tests	Rayleigh, piston ring, no rework, S/N 02	Floating ring, no rework S/N 04	Scheduled Inspection, LOX seal in excellent condition
13	633-665	, 33	72.6	LOX acceleration tests	Rayleigh, piston ring, no rework, S/N 02	Floating ring, no rework, S/N 04	Completed 10 hours with 300 starts on Rayleig LOX seal, LOX seal condition
14	666-749	84	249.7	LOX acceleration tests	Spiral groove, carbon, new S/N 04 housing, S/N 03, from build 8, mate lapped	Floating ring, seat, new, S/N 05	Scheduled Inspection, completed total of 10 hours with 300 starts on spiral groove LOX seal, both seals in excellent condition

## TABLE 7. LOX SEAL HARDWARE SUMMARY

.

	Seal Assembly, Turre	Scal Ring	Mating Ring Part No		
Build No.	Type Part No., Serial No.	Serial No., Lox/Turbine. Material	Part No., Serial-No-, Haterial	Hardware C Pretest	ondition- Posttest
1	Floating Ring R5009690X 01	Alterial 01/01 Carbon G84 Inconel X	RS009667X 02 Chrome/Honel	New	Good condition; carbon polished; no visible wear, mate had slight trace of contact
2				Same as build 1; no rework	Good condition; carbon recess pads worn 0.00001 to 0.00002 m {0.0002 to 0.0004 in.]. Mate had slight contract pattern; tester bearing failed
3				Same as build 2; no rework	Good condition; no wear; mate had slight trace of contact
4				Same as build 3; no rework	Good condition; carbon recess pads worn 0.00003 m (0.0001 in.); carbon polished mate had slight trace of contact
5				Same as build 4, no rework	Seal damaged by tester bear- ing failure, carbon severly worn and scored, end plate rubbed mate slightly; mate had grooves worn into sur- face at seal rings
6	Floating Ring RS009690X 04	04/04 Carbon G84 Inconel X	RS009667X 001 Chrome/Hone1	New	Lox carbon wear 0.000013 to 0.000020 m) (0.0005 to 0.0008 in.) Turbine carbon wear 0.000010 to 0.000018 m (0.0004 to 0 0007 in.) Hate slight traces of wear maximum depth 0.00004 m (0.000150 in.) on one side
7	Floating - Ring RS009690X 03	03/03 Carbon 684 Inconel X	* RS009667X 003 Chrome/Hone1	New	Seal severely damaged by dry spinup; carbon rings' broken, mate ring grooved by rubbing on housing
8	Floating Ring RS009690X 02	02/02 Carbon 684 Inconel X	RS009667X 004 Chrome/Honel	News	Seal damaged by bearing failure; carbon lift pads worn away; mate grooved by housing
9	Floating Ring RS009690X 04	04/04 Carbon 684 Inconel X	RS009667X 001 Ch <i>ro</i> me/Hone1	Same as build 6; 'no rework	Lift pads worn away; seal rings worn 0.000056 to 0.000064 m (0 0022 to 0.0025 in.); mate had contact pattern heavier on one side
10	Floating Ring R5009690X Q4	04/04 Carbon 684 Inconel X	RS009667X 001 Chrome/Honel	Lift pads worn away. Same as build 9, no rework	Carbon polished, no addi~ tional wear, heavy contact pattern on mating ring
11				Same as build 10; no rework	Same as build 10
12				Same as build 11; no rework	Carbon worn additional (0 008 in.) diameter on lox side and 0.00028 m (0.011 in.) diameter on turbine side; mating ring grooved 0.00005 m (0.0002 In.) deep on lox side and 0.00003 m (0.001 in.) and on turbine side
13				Same as build 12, no rework	Carbon chipped at edge of dam; carbon and mating ring worn, no additional wear since build 12
14	Floating Ring RS009690X Clevite 05	05/05 Carbon G84 inconel X	RSSD09667X DDl-I Replated Chrome/Honel	Seal new, mating ring replated	Lox side carbon ring worn 0.000043 m (0.0017 In.) diameter; turbine side carbon ring worn 0.00066 m (0.0026 in.) diameter, most of lift pads worn off; mating ring had heavy com- tact pattern, no wear

## TABLE 8. HELIUM SEAL HARDWARE SUMMARY

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	Seal Assembly, Type	Seal Ring Part No.	Piston Ring Part No	Mating Ring Part No.,	" 	
Build No.	Part No., Serial No.	Serial No., Material	Serial No., Material	Serial No., Material		2 Condition Posttest
					Pretest	
1	Bellows RS009844x 01	RS009697X 08 Carbon P692		RS009698X 04 Chrome/Hone1	New	Dam worn 0.000038 to 0.000051 m (0.0015 to 0.0024 in.); recess pads worn off; mate had irregula contact pattern with worn spots
2	Piston Ring RS-09849X 01	RS009697X 05 Carbon P692	SSCY5097X 05 Carbon P5NR2	RS009698X 01 Chrome/Monel	New	Dam and recess pads worn even 0.000005 m (0.0002 in.); mate had uniform contact pattern with one worn spot; tester bearing failed
3	Piston Ring RS009849X Ol	RS009697X 07 Carbon P692	SSCY5097X 01 Carbon P5NR2	RS009698X 02 Chrome/Honel	New carbon and mate; same housing, pilot ring and piston ring	Dem and recess pads worn even 0.000013 m (0.0005 in.); pilot ring chrome flaked at pilot; mate had uniform contact pattern with one worn spot
ų	Spiral Groove RS009695X 01	A28-0812-11 01 Carbon P692	A28-0812-12 01 Vespel SP211	C28-0812-10 01 Chrome/Mone1	New; installed wave spring around piston ring	Good condition; no rubbing contact :
5					Same as build 4; no rework	Seal damaged by tester bearing failure; carbon seal ring broken; face polished; mate' spiral grooves worn slightly on one side; housing rubbed mate slightly
6		A28-0812-11 02 Carbon P692		A28+0812-10 02 Chrome/Mone1	New carbon and mate, same housing, piston ring, wave spring around piston ring	Good condition; carbon face rubbed slightly; mate ring very slight rub marks; piston ring two slightscore marks on ID
7			¥		Same as build 6; carbon lapped flat; mate lapped clean	Seal severly damaged by dry spinup; mate ring rubbed agains housing and rotated on shaft
8	Spiral Groove R5009695X 03	A28-0812-11 03 Carbon P692	A28-0812-12 02 Vespei SP211	A28-0812-10 03 Chrome/Monel	New, wave spring around piston ring. Mating ring nicked slightly in dam area	Seal damaged by tester bearing failure; carbon ring broken, carbon surface polished; mate surface scored and rubbed
9	Piston Ring R5009849X 02	RS009697X 01 Carbon P692	SSCY5097-8 02 Carbon P5NR2	RS009698X 03 Chrome/Monel	New	Excellent condition; slight trace of contact on mate ring; carbon as-new
10			-		Excellent; same as build 9; no rework	Excellent condition; slight trace of contact on mate ring; carbon polished; no wear
11					Same as build 10; no rework	Same as build 10
12					Same as build 11; no rework	Same as build ll
13					Same as build 12; no rework	Excellent condition; slight trace of contact on mate ring; carbon face polished; no meas- urable wear; piston ring and adapter in good condition with no visible deterioration
14	Spiral Groove R5009695X 03	A28-0812-11 04 Carbon P692	A28-0812-12 -7456 Rev. C O3 Vespel SP211	C28-0812-10 03 Chrome/Mone1	Carbon seal ring new; housing and mating ring same as build 8; mating ring lapped to partially clean up scoring from previous rub, new piston ring revised to elimi- nate vent slot; wave spring around piston ring	Satisfactory condition; mating ring rubbed slightly; no signi ficant wear; carbon slightly scored from rubbing, piston ring in excellent condition

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## TABLE 9:LOX SEAL INSPECTION SUMMARY(ENGLISH UNITS)

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-	Spring	Spring	Dam Wear	0 (P	Recess Pa epth, inc re/Postte ift Pad N	h´ st)			ic GN <sub>2</sub> (Pre/Po Pressure		scim
Build No.	Load, pound	Rate, lb/in.	(Average), inch	1	2	3	25	50	100	200	300
1	4.9	200	0.0019	0.0005	0.0006	0.0004	66 770	110 1575	218 3025	455 10,540	659 16,330
2	2.86	54	0.0002	0.0005	0.0006 0.0004	0.0006	27 54	67 129	147 262	324 476	488 600
3	2.8	54	0.0005	0.0005	0.0006	0.0006	78 650	120 1100	145 1850	272 300	410 4200
			0	2	2	2					
4	6.1	88.7	0.0000	0.0004	0.0004	0.0004	1200 1075	2000 1700	3500 2600	11,232 5616	21,600 13,478
5	6.7	90.0	- 3	2 0.0004 3	2 0.0004 3	② 0.0004 ③	950 3	· 1450	2275 3	5616 3	14,515 ③
			0	2	2	2					_
6	4.7	91.3	0.0000	0.0004	0.0004 0.0004	0.0004	1720 700	2450 1275	3350 1900	6560 6480	15,900 16,000
7	4.5	92.5	() (4)	② 0.0004 ④	② 0.0004 ④	② 0.0004 ④	1880 (L)	2350 (b)	3310 ④	9330 (4)	19,900 (4)
			•	2	2	2				· ·	
8	3.7	72.5	3	0.0004 0.0004	0.0004 0.0004	0.0004 0.0004	2670 ③	3920 3	4230 3	11,060	22,810 ③
9	2.0	14.0	0,0000	0.0007	0.0006 0.0006	0.0006	102 186	168 333	364 494	730 740	1090 1350
10	2.1	14.0	0.0000	0.0007	0.0006	0.0006	95 98	220 210	518 429	1075 830	1550 1360
11	2.1	14.0	0.0000	0.0007	0.0006	0.0006	100 190	235 245	535 430	1075 900	1625 1525
12	2.1	14.0	0.0000	0.0007	0.0006	0.0006	75	150 266	300 341	900 738	1350 1163
13	2.1	14.0	0.0000	0.0007	0.0006	0.0006	12 75	100 115	364 320	880 690	1420 1020
14	3.9	85	0.0000	0.0003	0.0003 0.0003	0.0003	325 440	545 700	1080 1370	6700 7200	12,300 18,400
0	Carbon se	al ring f	ace wear	L		I	£	<u>I</u>	£	I	L
2	Mating ri	ng spiral	groove d	epth							
3	Seal dama	ged by te	ster beaŕ	ing fail	ure						
4	Tester da	maged by	dry spin	44 <b>-66-6</b> -6-6-6-6-6-6-6-6-6-6-6-6-6-6-6-6-				VUID-MIRINENT			

## TABLE 10. LOX SEAL INSPECTION SUMMARY

Bui 1d	Spring Load,	Spring Rate,	Dam Wear (Average),	i	ecess Pad Depth, m			Stat	ic Leakag Pressure	e m <sup>3</sup> /minute N/m <sup>2</sup> g	
No.	N N	N/m	m	1	2	3	172,369	344,738	689,476	1,378,951	2,068,427
3	21.80		0.000048	0.000013	0.000015	0.000010	0.0011	0.0018 0.0258	0.0036 0.0496	0.0075 0.1729	0.0108 0.2678
2	12.72		0.000005	0.000013 0.000008	0.000015 0.000010	0.000015 0.000010	0.0004	0.0011 0.0021	0.0024 0.0043	0.0053 0.0078	0.0080 0.0098
3	12.45		0.000013	0.000014 0.000003	0.000015 0.000003	0.000015	0.0013	0.0020 0.0180	0.0024 0.0303	0.0045 0.0492	0.0067 0.689
4	27.13		0.00000	② 0.000010 0.000010	② 0.000010 0.000010	② 0.000010 0.000010	0.0197 0.0176	0.0328 0.0279	0.0574 0.0426	0.1842 0.0921	0.3542 0.2210
5	29.80		8	② 0.000010	② 0.000010	② 0,000010	0.0156	0.238	0.0373	0.0921	0.2380
			` M	- 3 0.000010	3 2 0.000010	3	3	3	3	3	3
6	20.90		0.000000	0.000010	0.000010 0.000010	0.000010	0.0282	0.0402 0.0209	0.0549 0.0312	0.1076 0.1063	0.2608 0.2624
7	20.02		B	2 0.000010	② 0.000010	② 0,000010	0.0308	0.0385	0.0543	0.1540	0.3264
					<b>A</b>	4	4	(4)	4	4	4
8	16.46		3	0.000010	0.000010	0.000010	0.0438 3	0.0643 3	0.0694	0.1814 3	0.3741 3
9	8,90		0.000000	0.000018	0.000015 0.000015	0.000015 0.000015	0.0017	0.0028 0.0055	0.0060 0.0081	0.0120 0.0121	0.0179 0.0221
10	9.34		0.000000	0.000018 0.000018	0.000015 0.000015	0.000015	0.0016	0.0036 0.0034	0.0085 0.0070	0.0176 0.0136	0.0254 0.0223
11	9.34		0.000000	0.000018	0.000015 0.000015	0.000015 0.000015	0.0016	0.0039 0.0040	0.0088 0.0071	0.0176 0.0148	0.0267 0.0250
12	9.34		0.000000	0.000018	0.000015 0.000015	0.000015	0.0012	0.0025 0.0044	0.0049 0.0056	0.0148 0.0121	0.0221 0.0191
13	9.34		0.000000	0.000018 0.000018	0.000015	0.000015 0.000015	0.0002	0.0016	0.0060 0.0052	0.0144 0.0113	0.0233 0.0167
14	17.35		0.000000	0.000008 - 0.000008	0.000008 0.000008	0.000008 0.000008	0.0053 0.0072	0.0089	0.0177 0.0225	0.0199 0.0181	0.2017 0.3018
$\boxed{\bigcirc}$	Carbon se	al ring	face wear	<b></b>			<u> </u>				
2	Hating ri	ng spira	l groove dep	th							
	Seal dama	nged by t	ester bearing	a failure							
	Tester da	maged by	dry spin		······				•		

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### TABLE 11. HELIUM SEAL INSPECTION SUMMARY (ENGLISH UNITS)

Build No. (1	Ring OD, inch (Pre/Posttest)						R		d Depth, Posttest					(at 30	Leakage psig), im
No. (1		TIC/LO	ch sttest)		LOX Sid	e Lift P	ad No.		Tur	bine Sid	e Lift P	ad No.			sttest)
		LOX	Turbine	1	2	3	4	5	1	2	3	4	5	LOX	Turbine
	0.8129 0.8127	0.8143	0.8145	0.0010	0.0012	0.0012	0.0013	0.0011	0.0014	0.0016	-0.0015	0.0017	0.0015	4300 4060	2400 1840
2	() 0.8128	() 0,8145	() 0.8144	0.0006	(1) 0.0010	(1) 8000.0	() 0.0009	() 0.0008	(1) 0.0010	0.0012	0.6012	0.0013	0.0012	3970 4070	2460 2077
3	0.8128 0.8128	0.8145 0.8145	0.8144 0.8144	0.0006 0.0006	0.0010 0.0010	0.0008 0.0008	0.0009 0.0009	0.0008 0.0008	0.0010 0.0010	0.0012 0.0012	0.0012	0.0013 0.0013	0.0012 0.0012	4550 3150	3150 2700
4	0.8128 0.8128	0.8145 0.8147	0.8144 0.8148	0.0006 0.0005	0.0010 0.0009	0.0008 0.0007	0.0009 0.0008	0.0008	0.0010 0.0010	0.0012 0.0011	0.0012 0.0011	0.0013 0.0011	0.0012 	4200 2900	3600 2500
5	0.8126	0.8147 2	0.8148 2	0.0005	0.0009 2	0.0007 ②	0.0008 2	0.0007 ②	0.0010 2	0.0011	0.0011	0.0011	ē	3100 2	3950 2
6	0.8126	0.8143 0.8155	0.8143 0.8156	0.0007 0.0001	0.0005	0.0007 0.0002	0.0010 0.0005	0.0010 0.0002	0.0007 0.0000	0.0007 0.0001	0.0007	0.0007 0.0003	0.0007 0.0001	1250 3750	1650 4650
7	0.8123 3	0.8142 3	0.8139 3	0.0009 3	0.0011 3	0.0012 3	0.0009 3	0.0010 3	0.0006	0.0008 3	0.0007 ③	0.0007 3	0.0007 3	1300 3	1500 3
8	0.8130 0.8123	0.8141 2	0.8143 2	0.0009	0.0012	0.0009 2	0.0012	0.0008	0.0008	0.0008	0.0008	0.0010 ②	0.0007 2	900 9570	1880 6120
9	0.8130 0.8133	0.8151 0.8173	0.8154 0.8179	0.0002 0.0000	0.0000 0.0000	0.0002 0.0000	0.0005 0.0000	0.0002 0.0000	0.0000 0.0000	0.0001 0.0000	0.0001	0.0003 0.0000	0.0001 0.0000	3980 9750	4900 9500
10	0.8126 0.8130	0.8173 0.8175	0.8179 0.8180	0.0000	0.0000 0.0000	0.0000	0.0000 0.0000	0.0000 0.0000	0.0000 '0.0000	0.0000 0.0000	0.0000	0.0000 0.0000	0.0000	8068 4360	9085 5270
11	0.8130 0.8130	0.8175 0.8172	0.8180 0.8168	0.0000	0.0000	0.0000	0.0000	0.0000 0.0000	0.0000 0.0000	0.0000 0.0000	0.0000	0.0000	0.0000	7500 7200	8900 8900
12	0.8130 0.8130	0.8172 0.8261	0.8168 0.8278	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000 0.0000	0.0000 0.0000	0.0000	0.0000 0.0000	8750 	7750
13	0.8130 0.8130	0.8261 0.8260	0.8278 0.8277	0.0000 0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000	0.0000 0.0000	0.0000	0.0000 0.0000	13075 14000	17000 17500
14	0.8126 0.8126	0.8140 0.8157	0.8142 0.8168	0.0012	0.0013	0.0001	0.0005 0.0000	0.0006	0.0007 0.0000	0.0010	0,0002 0,0000	0.0003 0.0000	0.0012 0.0002	1500 3600	1150 2100

(3) Tester severely damaged by dry spinup

## TABLE 12. HELIUM SEAL INSPECTION SUMMARY (SI UNITS)

	Mating Ring OD, m	1	Ring		(P	Seal Rir ID, m re/Post Te					Rece	ss Pad De	pth, m		Hellum	
Build	(Pre/ Post	(Pre/Post				LOX Side	\	i	Build		Tu	rbine Sid	le	1	(at 20	
No.	Test)	LOX	Turbine	1	2	3	4	5	No,	1	2	3	4	5		inute
1 2	0.020643	8		0.0000251		8		0.0000279 (1) (1) 0.0000203	2			8	0.000043		0.0666	0.030
3	0.20645 0.026045	0.20688 0.20688		0.0000151			0.000023	0.000020 0.000020	3	0.000025	0.000030	0.000030	0.000033	0.000030 0.000030	0.0746 0.0517	0.051 0.044
4	0.020645 0.020645			0.0000152			0.000023	0.000020	4				0.000033		0.0689 0.0476	
5	0.020640	0.020693	0.020696 2	0.000013	0.000023	0.000018	0.000020	0.000018	5	0,000025	0.00028	0.000028	0.00028	Ō	0.0508	0.648
6		0.020714	0.020716	0.000018	0.00000	0.000005	0.000025	0.000025 0.000002	6	0.000000	0.00003	0.00003	0.000018	0.000003	0.0615	0.076
7	0.020632	0.020681	0.020673	0.000020	0.00028	0.000030	0.000023	0.000025 3	7	0.000015	0.000020	0.00018	0.00018	0.000018	0.0213 3	0.024
8	0.020632	2	2	0.000023	0.000030	0.000023	0.000030	0.000020	8	2			0.000025	2	0.1569	0.100
9				0.000005		0.000005 0.000000	0.000013	0.000005 0.000000	9				0.000008			
10	0.020640 0.020650				0	0 0	0	0 0	10	0.000000	0	0 0	0	0 0	0.1323 0.0715	
11	0.020650 0.020650				0 0	0 0	0	0 0	11	0 0	0	0	0	0 0	0.1230 0.1181	
12	0.020650 0.020650			0 0	0	0 0	0	•0 0	12	0	0	O C	0	0 0	0.1435	Q.127 
13	0.020650 0.020650			0 0	0 0	0 0	0 0	0 0	13	0	0	· 0	0	. 0 0	0.2144 0.2296	
14		0.020676 0.020719		0.000030 0	0.000033 0.000005	0.000003 0	0.000013	0.000014 0	14	0.000018 0	0.000025		0.000008 0	0.000030 0.000005		
() ()		iged by te		ing failur			L		<u> </u>	L	I		I	I <u></u>	1	

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## • TABLE 13. DATA SUMMARY (ENGLISH UNITS)

TEST TIME	SECED			LOX	SCAL				PELION	SFAL	
NO. MIN.	493						1218465	11.25 00.		1175 TCM	· FARACE
-			_علية £ليد. S£4L	LP/9.P3± DR&I4	_U/S 11 41	DRIN		PURGC	DIS PR.	PURGE	TOTAL
		0120	0.14	PSIG	f	F	SCEM	PSIG	GRAIN PSIG	<u> </u>	<u>sc</u> fm
		67.5	64.0	5.00	25 25	42	.26	35.0	.70	73	5.23
35 .1C3 36 .283	***** 342	67.5	65.2	4.27	37	16	.26	37.09	1.40	70	4.60
7	324	68.5	62.2	3.90	29	18	•2 <b>0</b>	30+4	1.40	70 70	4.70 4.70
38 .050	322	67.5	62.6	4.2	20	12	.23	32.4	1.36	70	- 4.70
39 .(75 43 6.403	32013 25301	63 C £6 S	62.4	4.20	-35	17	_(2).51_	32.0	.90	<u> </u>	4.80
1 .285	-37657	152.5	143.2	14.27	65	1	•69	31.5	1.52	67	4.00
42 .CA7	37665	151.5	143.2	15.20	63		.78	31.5 32.5	2.20	<u> </u>	3.70
43 .250	48005	152.F 151.f	142.8 147.8	16.29 >23.00	62 60	,	*****	31.2	3.70	67	2.90
45 .152	65 35	152-5	143.2	>20.00	60	2	****	31-3	3.40	61 87	2.70 4.80
46 +100	1500	146.5	142.8	3.8	<u></u>	25	<u>.17</u> .23	31.5	<u>.44</u> ,96	89	4.63
47 .113 48 .225	27069 36535	152.0	143.2	4+31 4+31	R3 *2	31	.29	29.9	1.44	89	4.10
48 .225	58:0-	157.5	143.2	12.90	79	27	1.01	31.5	4.90	87	3.90
50 <u>.3"</u> J	8752"	153.5		>?:.!	63	<u>32</u> 27	***** 6.36	31.3	<u> </u>	<u>87</u> 89	2.70
51 .250 52 .300	8650 92030	15345 16040	140.L 144.0	15.60 2.99	84 84	15	6.55	30.3	5.32	87	4.40
52 .377 53 2.103	76752	153.5	142.6	1.20	86	3	2.92	32+6	90	99	4.90
54 6.900	8942~	156.5	144.0	2.35	85	56	3.22	<u>_</u>	<u>5.40</u> 7.18		<u>9+60</u> *****
55 . 93.	2437 911	152.C ¢*****	152.:	******	76 \$48#\$4	******	****** ******	*****	****		7\$\$ <b>\$</b> \$ <b>\$</b>
56 .423		163.0	157	3.16	75	*****	49700*	32.2	9.96		44444
58 2.037	8975	16.7.7	157.5	1.94	£		<u> </u>		<u></u>	87 95	<u>*****</u> 3.91
59 .672	854	154+0	152.0	1+36	¢7 ¢2	******	5.61 5.51	29-3	7.70	92	3.93
67 .953 61 6.383	8651C 82757	153.5	153.0	1.30	¢7	*****	5.95	31.0	9.52	94	.3.95
62 .230	915:"	******		******	****	******	*****	*****	****		******
63 .322	<10 >	*****		A000000	202222 202222	******	******	******	******		******
54 .33 <sup>-</sup> 55 .472	913.1 912.1	*****		******		******	*****	*****	******		*****
65 2.150	936.35	203.0	199.5	2.54	75	55	7.19	32.0	9.12	95	3.81
67 6.353	92031	333+C	202.0	5.00	74	82	7.06	31.8 29.5	9.00	92 91	3.80 3.42
68 6.4°C	90107	253.0	245.0	2.98	<u>74</u>	52 ******	7,84	35.0	96	****	3+64
65 6.470 72 6.400	96020	372.7	297.1	3.39	*****	****	2.22	30.0	9.10	494 594	3,65
7. 6.401	92112	3_5	799.:	3.50	*****	******	2.55	32-2	9.25 8.76	44444 42444	3.63
72 6.452	E9600	357.0	257.5	3 24	\$\$\$\$\$\$ \$\$\$\$\$\$	*****		$-\frac{31}{21}$	35.8	*****	3.78
75 6.45	89923 69911	295.0	268.5	>22	75	75	14.35	31.5	8.04	89	3+84
75 .500	662.^	295+2	289.5	>25.23	74	75	14.15	32.1	8.0	97 85	4.65 3.06
76 6.303	898	127.45	<u></u>	12.78	72	******	\$.63 7.38	32.5	8.88 9.18	83	
77 6.20	91010	301.0	292.0	.6.	87	-57	12.52	29.3	.65	110	3.40
79 .17:	1985	157.2	159.5	.60	<b>P</b> &	-55	12.53	29.5	1.50	110	3.4J 3.90
8: .210	34027	187.5	195.0	1.2.	86 <u>85</u>	-45	1,43	29.5	2,50	115	3.40
83 .380	4875 71125	155.2	163.C 369.C	7.	≏⊃ 89	-43	9.03	29+6	9.30	104	
83 .752	628uC	149.0	155 C	•50	90	15	9.00	30.3	7.19	111	
84 .173	52000	145.0	153.0	•67	90	27	19.40	28.5	5.20	113	
85 3.3(3	<u>- 27' . r</u>	151.5	<u>157.C</u> 173.C	<u> </u>	10 87	<u>20</u> 43	<u>9.47</u> 5.93	29.5	9.30	113	3,60
87 4.213	70100	161.5	155.5	.97	92	45	5.70	20.0	9.36	<u></u>	3.53
38 6.223	2263	164.7	:69.5	.9	÷2	52	6.7^	29.5	9.3ű totsztag	112	3+50 ******
87 .150		*****	168.C	****	****** 96	<u>****</u>	2.20	****** 27+4	9.63	1 79	3.89
90 6.350 91 1.503		165.0	162**	.7.2	<u>91</u>	4	2.47	29,+7		103	3.90
07 6.NET	207**	157.5	161.7	.50					10.50	100	3.40
93	****	200200	******	****	*****	\$\$\$\$\$\$ \$\$\$\$\$\$	<u>000000</u>	C00\$90	******	*****	0040044
94 ,100	******	****** ******	******		\$\$\$\$\$\$	*****	*****	******	*****	444¢¢\$	*****
96 .520	8:5:5	152.	154.5	***	¥2	-7	2.13	33+2	2 - 12	85	1+73
97 6.022	78051	153.5	15.5			*****		29.7		59 <u>.</u>	5+30
98 4031 199 - 119	******	****	****** *****	*******	005000	******	******	007800		480404	*****
3.770	834,	223	17840	1.32	£5	Z*	2 4 4 4	2 Y Y Y	2.60	62	5.23
01 3.003	79:50	202.5	<u>157.i</u>	1.30	67	44	3,-2	30+0	2.52	85	<u>5,36</u> 5,50
32 .500	729:"	25240	24940	1.87	60 	22	3.90	29.5 29.2	2.10	192	5.42
0 <u>3 5.000</u>	116	2 <u>&gt;307-</u> 0	299	2.20		51	4.32	29.7		89	5.40

	1140	SFEED	********		<u> </u>	SC4L				_FEFINH	SEAL	· · · · · · ·
40+	HIN.	89 <i>4</i>	075 P2.	375 PF.	P/5 PC+	U/5 15-P	DIS TEHP	LEAFAGE	BZS PR.	_D/S PR.	U/S TEMP	- I E AVAG
			C & V I I Y	STAL	ora IM	CAVITY			PURGE	TURE SIDE	PURGE	TOTAL
			<u> </u>	<u>P-1</u> C	PSIG	<u> </u>	<u> </u>	<u></u> \$ <u>CFM</u>	<u>PS16</u>	FRAIN PSIG	<u> </u>	<u>SC</u> FH
125	6.003	773:0		300.0	2.07	61	53	4.40	29.0	2.62	85	
105	6.003	775.35	335.0	303.0	2.02	<u> </u>	55	4.41	29.2	2.68	85	
107	-850	71530		360+5	2+40	70	ų	4.73	31.3	2.50	82	
109	6.017 6.003	77437 78720	<u>305.0</u> 375.0	293.C	2.20	<u>60</u> 56	4e 48	4.53	33.3	<u>2+88</u> 2+92	81	
113	6.000	77325	310.0		2:47	56_	48	4.90	32,2	2.86	- 79	
111	6.000	77425	295.0	279.C	2.10	56	44	4.43	34.5	2 94	80	6.60
112	•250	32205	215.5	<u>192 • C</u>	<u>5+20</u>	-280	-297	6•50	28.8	6.65	85	
113 114	+450 +250	7150" 2220"	249.7	212.C	13.87	<b>****</b> **	~28" ******	15.22 ******	E*35 \$***	2.18 ****	82	2.93
115	.25.	567	****		******	*****	******	******	*****	******		******
116	.187	78265	*****		*****	******	*****	*****	*****			******
117	.753	23765	142.5	121.2	.32	~267	-202	5.90	30.7	-48	82 82	
118 119	<u>.503</u> -430	66705 2915	207-0	172.0 55.0	<u>5*02</u> *82	-275	-213 -175	<u>9.40</u> 3.90	<u>30-9</u> 29.7	<u>1.94</u> .58	81	
120	.442	64230	95.0	85.0	.92	-255	-187	3.70	30.0	1.88	81	2.00
121	.225	2551	141"	114.0	2.70	-275	-187	6.50	29.0	£0	78	
122	.CE3.	24521 64501	92.C	****** 85*C	****** 1+4C	<u>**</u> **** -255	-180	4.70	<u>******</u> 3.~+9	1.93	****** 78	********
123	1.453	647CD	98.0	85 . L 95 . C	1+40	-255	-183	3.40	29.4	2.50_	78	
125	223	57420	75.0	44 . C	.60	-257	-133	4.20	29.7	1.60	75	3+30
126	. 78 .	28501	143.0	122.5	1.73	-2€7	-149	5.00	30.4	-60	75	
127 129	.255	5923° 6753°	******		*******	4***** -28G	******	******	****** 30.4	2.50	¥***\$ <b>*</b> 72	******* 3,20
129		4330-	73.5	94 2 67 7		~267	-133-	-3.20	24.8	1.50	86	
130	1.120	66533	117-0	<u>112-</u>	1.54	-272	-140	4.33	35.7	. 2.10		
131	,723	6640	112.2	*****	.1.23	-265	-127	4.30	33.3 30.0	2.30	69 75	
132	2.085	65835 41230	<u>121.</u> 69.	<u> </u>	.13	-197 -257	-97 -112	2.33	30.7	2.20	70	
134	252	674.15	******		*****	*****	******	******	*****	*****		******
135	,53.	67225	169.3	115.2		-267	-133	5.20	29.6	2.30	. 70	
136	<u>.28:</u>	67735	115.	132+2		-267	-122	<u>4.87</u> _	29.3 30.8	<u>2.30</u> 2.30	70	
137	.337	65527 65527	104.0 120.0	93.C 128.C	1.40	-255	-125	4.6C 4.10	30.2	2.20	65	
139	+673	676	122.0	155.2	5.12	+275	-167	. 8.87	31.7	2.30	63	
.14		6810	*****		******	******	<u>****</u>	******	<u>********</u> **	2.50		******* 5 •2(
141 142	1.83.	66905 667.5	126.5	114,2 112,5	1,42	-272	-135 -130	4.40 4.30		2.50	65	
143	<b>7.</b> scc	6352-	110.0	107 .0	1.50	-26 3	-122	4.60	29.2	2.50	66	5.5
144	1.430	31400	129.0	117.0	1.60	-266	-122	4.67	31.3	•93	67	
145	2.472	2982- 3100-	127.0	134.C 96.C	1.3"	-275	-115 -145	3.70	32.3 35.7	•90 •90	63 65	
146		31857	108.0 108.0	97.0	1.47	-215	-140	3.60	30.8		65	
148	5.03.	<u>596.</u>	156.0	144.5	.51	-170	-101	1.60	3;,0	1.70	65	
149	5.43_	\$82:00	134.0	122 · C		-185	-72	1.85	30.4	1.40	67	
<u>155</u> 151	.37.	7715- 8316-	183+2	165.0	4.55	425593 435553	-152	7+57	36.4	3.70	65	6.9
152 152	• 13	\$11CT			*****	******	******	\$ 3 6 5 5 5 5	******			******
<sup></sup>	1.655	8250	171.0	156.0		-275	-15Z	1.82	30.46			6.7
154	1.520	80400		157+0	•5 <sup>^</sup>	-167	-75	1.50	29.8		68	
. 155	2.417	<u> </u>			3		-57	++++++	2° 48. ****			5+5
156 157	+13C 4,150	844:1 830:50			******	******			30+2			0.7
158	• 125				*****			*****	\$\$\$\$\$\$	****	****	*****
159	5.732				3	-57		<u></u>	34	3+67		6.8
163	.072	267:" EE7:"	******	******	*******	 \$\$\$\$\$\$ \$\$		*****			*******	
_161_ 152		<u>88627</u> 85001	******	*****	3000000	******	******	*****	*****	******	*****	* ******
163_	1,053	222.1	233.0			237	-6.7		31.2			7.8
164	•721	29615	164.0	139.0	1.10	-272	-82	3-60	29.0		64 E 4	
165	<u>.47</u>	<u>31021</u> 28930		147.	2,57	-276	<u>-1]1</u> \$*****	<u></u>	29.3			* ******
166 _167	-221 1.115				4.52		155					
_16 S	.230	8410 <sup>r</sup>	254.0	235 . 3	5.10	-275	-140	5.10	31+2	4.90	64	8 41
<u>_</u> 169_		<u>P+3C</u>		23 <u>5 k</u> .j	4+67	275		7 • 7 1	31+C 35+5	4+90	6 <u>4</u> 64	a+3 8+3
176	+45J	84D37	254.0 251.0	235+0	4.27 <u>3.6</u>	-275 275	-176	4 63				
172	2.352	<u>838.</u> 255.5		233.44	3.50	~275	-147	6.42			66	5 5.4
		83637		272 • 6			13°	מי כ			66	8.9

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TABLE 13. (Continued)

		SFELD		· · · · · · · · · · · · · · · · · · ·	<u>10x</u>	<u>ŞÇ4L</u>				PELIUR	SFAL	
0.	HIN+	884	11/5 62			-				DIS PR.		
			C 4 V ]-1 Y-	'SFAL	DRAIN	CAVITY	DRAIN	<u>, , , , , , , , , , , , , , , , , , , </u>	PURGE	TURE SIDE	PURGE	TOTAL
			<u> 112 -</u>		PSIC	٢	<u> </u>	SEEH	PSIG	GRAIN PSIG		<u>\$C</u> FM
75	1.050	83620	295.	279.5	4.33	-272	-135	7.33	29.0	4.80	66	8.10
16	.373	63765	292.2	271.5	4.51	-272	-147	7.53	29.0	4.80		
7	.582	e35.2°	285.0	265 . L	4.5.	-272	-140	7.50	29.9	4 • 8G		
18	1.(70	84100 83610	285.0	286.C	<u>4+30</u> 5+30		-137 -177	8.85	<u>35.5</u> 31.5	4.90		
9	1.183	62001	413.0	768.0	4.50	-275	-157	8.27	30.0	2 . 80		
ŭ	1.480	74527		401 • C	4.37		-152	7.80	30.0			
32	.475	84415	325.0	222.2	5.93		-162	<u> </u>		4.93 4.90		
33	1.183	845LC 84537	301.C 303.C	250.C 252.C	- 4 <u>+20</u> 4+33		-155	7.70	29.6 31.D			
34	.403	844.33	346.0	265.0	4.32		-147	7.92	32.3			6.2
36	2.823	84100	300.0	265.0	4.37	-272	142	7.62	<u>31.C</u>			
87	.150	19302	169.5	154+5	5+72		-293	18.12	29.6 *****	49 <b>.</b> ******		4.CC ******
88_	•127	21620	****** *****		******		*****	*****	74244			
89 90	.10.	24501	******		******		*****	*****	*****	******	*****	*****
71	.189	3850	235.	177.5	6.9°		-3.30		32.0			
92_	+203	41927	261.5	194.5	7.60		-295	22.32	32.0			
93	• 275 • 30"	48007		232.0	9.43		-293 -293	26.30 26.33	33+4	2.90	97	
94 95	•250	53254	333.5	252.5	7.02		-291	23.40	33.3	3.0	87	2.80
96	.ter	7650 7	******	*****	*****	*****	*****	£\$\$\$\$	*****	******		*****
97	• 2 C J	E17 J	*****		******		*****		*****			******
98 99-	.252	514CF 75450	322.5	235.5	9.20		-29:	26.62	31.8 35.3	<u> </u>		
37 30	1.850	757 01		702 0	2.70		-2.36	12.13	36.0		86	
оŤ.	•903			295.0	2.67	-282	-226	12.37	33.7			
22	2.955	78235		293.6	2.8?		-173		37.0	5.60		
3	.159	131un 24300	236.C 250.5	173.8 178.C	6.4 8.30		-300		30.4			
04 05	174	-32462		182.0						5.95		12.20
06	.699	76805	371.0	162.8	5.61				35.1			
07	1.357	75757		64.8	4+6				30.9 30.3			
28 0'9'''	•210	8263° 8253°		154.E	17.20				31.4			
12	.300	795.1	381.5	124 0				53.50	31.7	1.90		
11	3.127	74402	375.0	184 • C	6.20			19.22	3".0			
12	1.134	17560		164.5	<u>6.53</u> 15.23				<u>20.6</u> 30.2	- 1.90		
13 14	•247 •150	4125 3910	310.0	193.2 192.i	14.87				29.8			
15		686		197.2	23.33				30.1			
16	.285	74613	388.5	197.2	20.00	-295				1.75		
17	• 339	794.00		205.2	14.90				32.3 32.8		-	
18 19	1.149	7572		2 4 4					31.5			
22	.252	61207		274.8	15.00	-295			31.6	1.50		
21	+255	79730	422.5	207 . 2					31.5			
22	.228	<u>90001</u> 80600		257+2					30.0			
23 <sup>-</sup> 24	•183 •233	74400		207.6					29.8		8A 1	15.3
25	.212	02221	*****	2:7.6	+ 2.27	-291	- ~235	9.90	2			
25	3.067	75300		258.0					30+3			15.49
	3.158	76251	<u>704.</u>	279.5	3.5 <u>7</u> *******			2:.37				*****
28	<u>~,22</u> ?	86421	******		******	******	******	34C 1 04	0000000			
3Ĺ	.292	46701	.9	26								14.6
31_		76901	<u> </u>	325.0	7.5						*****	14.60
32 33	.22.	7762*	. ******* . ******								<u>\$</u> \$\$\$\$\$	
34	.217		******	245.5	14.50	,	-297	39.44	31.1	1.69		
35	.232	E30L0	*****	-14.0	9.5	-223		27.36				
36	3.000			329+1								14.6
37	<u>3.15</u> 0 3.082						-220	32.72	31.4		102	14.62
39	3.000						1-217	33+22			<u></u>	14+76
43	• Z Ž C	944	*****	*****		* *****	******					15.00
41	3.15.3		372.0	283.5							103	14.68
4 Z	3.150							3\$.86	31.4	2.20	152	15.04
						-285				2 • 20		15.05

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<u>1231</u>		SFEED			<u> </u>	SCAL				_FELION	SFAL	
NO.	MIN.	RPP	075 87.	1/5 PC.	. P/S cc.	_0/5 184P	0.45 TF H				U/S TEM	
			CAVIIY	SΓAL	DEVIN	CAVITY	DRAIN		PURGE	INDE SIDE	₽ប្ខRGE	TOTAL
<u></u>			0129	<u></u>	P5:6	<u>۲</u>	<u> </u>	<u>ŞCFM</u>	<u>PSIG</u>	GRAIN PSIC	<u> </u>	M3 <u>22</u>
245 <sup>55</sup> 3 246	1.65 .350	776.20	366.5	792 5	13.57	-285	-223	37.15	31.3	2.26	102	15.00
	1.125	77857	372.5	296.5	12.30	-260	-240 -212	36.38	31.0	2.13	102	15.0
	3.120	78227	368.5	298 . 5	12.27	-225	-217	32.74	31.2	2.10	99	
	5+15U 5+152	7733: 77120	368.0	295.0	11.69	-288	-217	32.23	31+0	2.10	97	15.20
	5.125	75622	367.5	295.5 287.5		-290	-225	$-\frac{31-71}{35-61}$	31+0	<u> </u>		15.2
	s.150	76500	360.0	271.0	13.42	-285	-233	32.69	31.0 31.0	2 C 2 . 20	97 109	15.20
	\$.153	76530	363.5	284.0	11,80	-285	-217	32.19	31.45	2.10	97	15.1
254 3	•45,	75851	359		12.5	-2/3 \$\$\$\$\$\$	+235	39.^6	31+2	2.00	96	15.13
256	.220	82011	*****		t\$\$\$\$\$\$	*****	******	******	******	*******		******
	5.253	7500	357,5	.67.*	11.37	-290	-235	37.45	31.4	1.90	90	15.29
258 259	.183	8100	20000C.		C9C0000	000000	000000	******	C\$4\$\$	******		******
260	•182 •369	90605 91107	400000 400000		*****	******	000000 00000	******	******	*****	¢¢¢¢¢¢	
	1.093	7730	385.0	276.2	15.77	-261	-255	45.14	****** 32.4	2,20.	******	14.72
	5 <u>5 92</u>	7760	390.5	261.5	14.37	-277		44.52	32+2	2.23	9.8	14.61
	1.193 1.263	77950	384 .0	285.5 281.0	12.00	-293	-246	35.24	31.3	2.20	96	15.02
	-123	779.2	395.5	253.5	12,63	-285	-253	39.44	29.7	2.10	86	<u>15.00</u>
66 3	\$•760	7625 C	381.5	282.0	12.40	-282	-251	47.29	31.3	2.20 2.10	87 88	15.01
	4.050	76857	287.5	282.5	12.32	+5°1	-240	39.81	31,6	2.10	90	14.98
	<u>5.59</u> ^	763	384.0	275.5	13.72	-290	-262	45.18	<u>32,5</u>	<u>2.2(</u>	<u>ş</u>	15_01
	4059	78300	389.0	223.5	9*77	-28\$	-236	44 82	32.6 32.5	2.10	89 90	14.46
	4.125	7687,2	392.5	305.0	8.30	-283	+223	33.70	32.5	2,20	92	16.55
	.090	776.2	344.0	135.5	7.90	-283	-235	33.76	32.7	Z • 20	<u>91</u>	16.48
179 3	• 281 • 250	757. 767.]	382.0	291.5 295.C	9.60 5.93	-28 3 -286	-255	39.71 33.13	33.3 33.2	2.20	91 69	16.43
75 3	.080	758:"	182.5	208.5	13.32	-285	-276	43.00	37.8	2.30	88	16.41
	• 2 55	758	382.5	268.5	12.50	-256	-277	44.11	33.8	2.30	89	16.93
	283+2 1100	76100	385 .C	267.5	12.23 8.10	-285 -285	-237	48.44	34.5	2.32	90	16.59
		779	381.5	299.5	8.37	-293	-257	38.93	33.7	2.20	90	16.93
83 3	.35"	760.0	352.0	275.5	13.97	-282	-267	55.13	34,5	· 2.20	90	16.16
	1.5.50		134 - 2	285.5	9.63	-355	-272	44.51	33.4	2.20	90	16,32
	. <u>Ce</u> n .Cs.	76235	*79.5 ·385.0	295.5	8+37	-285	-251 -253	<u>39.71</u> 37.47	37.4	<u>2•2C</u>	87	16.41
		765.3	775.5	281.5	10.42	-285 -	-28	45.57	32.7	2.26 2.30	88 80	16.40
	-222	77230	361.5	295.5	9.4	~266	~243	38.71	32.9	2.20	95	15,21
86 3	-181	777 _^ 7680	392.5	299	7.32	-284	-234	35.59	32.6	2.30	91	15.21
	115	78330	383.5 385.0	293.5 322.5	7.70	-283 -282	-234 -234	35+15 34+71	32.6 33.0	2.30	, 91 91	15.23
89 3	.25.	75912	383.5	294 .0	8.17	-28		37.27	33+2	2.20	91	15.25
	.15	756	380.0	285.0	15+70	-285	-287	44 44	33.9	2.40	88	15.20
	.:50 .:51	75801 78500	377.: 381.5	202.05	11.62	-265	-292	48.5	34-2	Z . 4ú	\$8	15.19
		7692	376.5	284	10 1	-285 ~285	-261	44.87	33.7	2.40	- <u>88</u> 09	15.29
	-100	78167	375.3	295.5	9.9	-283	-253	42.56	33.7	2,40	90	15.20
	•1:2	7812	373.5	295.5	7.9	-283	-242	47.91		22	9C	15 - 20
	.053	75520	382.0	284.C 287.5	11.4.	-2P? 262	-285 	47.31	34 8	2.3C 2.4C	91	15.22
	. <u>51</u>	750.5	275 5	392.5		-296	-255	41,07	]9_5 33.8	2.4C 2.30	<u>91</u> 87	15.31 15.53
	. 380	76335	376.5	297 5	9.41	-207	-255	43.^3	33.7	2.30	F7	15.45
20 2	.9.2	773.5	372	292.5	7+22	-297	-241	36.51	33.5	2.20	87	15.54
	• <u>1-2</u> 5 ( ) -	770_3 773:5	37:40	789. 299.5	7.3	-226	-2443_	34.51. 38.72	33.5 32.5	2_20 2_36	F8 86	15-52
	•[5]	761.	375 -	281.2	12 9	-294	-29	99.77	23.3	2.40	87	14.66
04 3	.5.53	758 35	179 . <u>-</u> 375	296 - 1	9.40	-293	302-	44.53	32.3	2.40	87	14.77
35 3	<u></u>	769_"	375	291+5	<u>8.9</u>	-787			32,2	<u> </u>	<u> 87</u>	14.79
136 3 137 _	•CE5 •15	768.) 	767.1	?{5." ******	7.5"	-2=2 *****	-240 #0000#	79.96 ******	32.5	2.3C 	97 ******	14.82 ••••••
	1982	77532	374.5	2.562	3.27	-215	+244	39.12	35.5	2.20	25	16.24
0,73	• C ::	77100	372.02		7.3 _	- 29 4		37.51 .	37.1	2	£ 6	. 17,33
10 3	1.05	762.	272.0	789.2	8 R.	-293	-275	45.31	32.8	2.30	66	14.60
	12	77620	378.2	71.4 0	<u>6.6</u>	<u>-2^2</u> -2F 3	<u></u> 245_ 243	39.95 38.28	<u> </u>	2.30	<u></u>	<u>14.7</u> 14.59
12 3 13 3	. <u>0</u> 57_		374.0	286 • 5	11.8^	2PJ	~?5 ·_			2.82	87	
14 3	Lins."	77225	372.0	277.5	13.41	-21 +	-296	55.84	33.2	2.80	23	14.49

TABLE 13. (Continued)

	<u>3411 -</u>	SFEE			_Lex	_SÇ\$L				PELINK	SFAL	
NO.	HIN+	864	1175 P.7.	4175 P.C.	0/5 00	11/6 784D	0.45 TEMP	•		_0/S PR		• F1280
			YTIVAD	STAL	DRAIN	CAVITY	ORAIN		PURGE	TURE SIDE	PURGE	TOTAL
			<u> 1510</u>	PTIC	PSTG	<u>r</u>	F	SCEN	PSIG	GRAIN PSIG	F	<u></u> SCF,M
315	3.120	773:		27 . L	14.10	-286	-293	54,03	33.5	2.70	83	14.45
316	3.052	776) 7822		283.5 285.5		-285	-292	54,65	33.2	2.70	<u>84</u>	14.44
317 318 .	3.080	7767		292.0	12.17 8.70	-283	-289	53+42 43+97	31 A	2.60	84	14.69
317	.30.	7470		28:	1.57	-277	-35.	22.24	32.1	1.10	86	14.86
32.0	.212 .382	55AC	<u> </u>	<u> </u>	2.67	-776.			32.7		<u></u>	14.89
321				358.5	7.42	-277	-257	32+93	3°-8_	2+50 •70	86 86	15.07
32 <u>2</u> 323	1.1:2	<u>6670</u> 7830		713.5	2.20	-272 -242	-271	21,20	32.5	2.40	83	15.16
324	3.679	7725	365.0	781.5	13.2 <sup>n</sup>	-282	-286	52.34	35.3	Z,63	41	15.41
325	3.080	779.		291.5	119	-282	-277	44.71	34.5	2.50	80	15.51
326	3.( 57	7763		267.5	11.23	-2°5 -285	-292 +296	47.73 50.88	34.8	2.50	<u>77</u> 11	15.48
327 328	3.092	7673		285.5	12.10	-284	-292	49.37	34.9	2.40		15.48
329	1.7	7686		377.5	11.32	-278	-296	49.02	32.9	- 2,30	78	15.73
33.3	100	1030		731.0	12.43	-273	~298	53.13	72.9	******	78	15.76
331 332	152	£290 *****		·1. •1.	247	-1.3	-286 *****	78.8E \$*****	39.6	4.20	74	13.82
	*****			297.5	2,74	+255		******	31.6			*****
334	1222	MCSSC	388.0	371.0	1.54	-257	-17	******	33.9	<u>, 56</u>	83	******
335		111223		429	1.92	-272	-51	*****	31.8	•36	86	******
336 337		<u>x10700</u>	7 412+5 C 432+5	423.5	1.99	-265	-61	******	27.5	<u>.38</u> 	<u>86</u> 87	******
22, Z38	1.93	M 2220	5 405 L	427.5	2.64	-268	-89	******	27.2	+ 22	87	-
339	292	92.7	C 446+5	426.0	2,78	~268	-78	******	28.2	"18	67	******
3 <u>9 )</u>	.143	563		344.5	2.86	-277	-131	*****	32.4	<u>+16</u>	83	*****
3 <u>4</u> 1 342	100 160	9270 5481		437.5 344.3	3.16	-271	-160 -122	******	31.9 53.0	•20 •18	87 85	******* ******
342	3.144	748		342.0	5.50	-283	-300	31.30	5.9	,24	82	9.21
344	100	9285	0 439+6	447.5	2.35	-256	-99	12.85	29.0	• 30	90	8.71
345	2.683	775		345.5	5.24	-282	~296	32,72	32.2	128	90	9.16
346 347	- 28:	9752 9033		422.5	2.62	-261	-86	15.95	28.3	<u>,28</u> ,28	<u>90</u> 91	<u>8.83</u> 9.23
348	282	657.3		348.5	4,49	-2PC	-178	23.65	31.6	.32	91	9.50
349	.17.2	283J	429.6	445.5	3+19	-244	-127	17.17	33.0	. ,28	87	10.35
350	3.031	7665		343.5	5.55	-227	-323	31.89	<u> </u>	.34	·: 88	12,49
351 352		>10000		425.C	1.53	-263 -261	-63 -35	4.39	27.6		86	9.02
353	.12	10000	419.5	452 . E	2.12	-256	-74	14.69	32.4	,18	57	9.99
354	.2"3	7775		259+7	5.35	-277	-155	27.47	32+4	+16	87	10.44
355	467	7693		349.0	5.29 4.74	-283	-296	29.13 29.57	30.5 31.0	.24 .22	E7 87	10.10
356 357	3.272	7482		<u>350.5</u> 353.5	5.20	+287	-324	29.9	35.7	.22	- 83	10.88
358	3.093	7520	2 397.5	264.5	4 . 64	~267	-300	29.25	31.7	.18	63	11.24
359	3.133	7520	5 397.5	364.5	4.74	- <u>2</u> P5	+322	28.73	31.7	.22	44	11.25
360 361	3,323	7630		<u>753+C</u> 759+C	5.32	-282	+296 -330	33.32	31.0	•24 •22	87 83	11.24
362 362	3,150	7590		358+0	4,54	-287	-204	28.27	31.0	.14		11.29
363	3.182	7530	391.0	258.5	4.44	-286	-302	28.54	32.8	.18	84	11.31
364	3-100	7550		357.£	4.74	-285	-37 -299	29.26 27,86	30.7 <u>30.7</u>	.22	85 86	11+33
36 <u>5</u> 366	<u>2.88:</u> 3.1:,	7570	<u>° 169 °</u> 392.5	<u>356.5</u> 355.5	4.46	-203	-299	28.17	51.1	<u>,22</u>	85	11.36 11.38
300 367	3.157	7610		355.F	9 • 57	595-	-297	28.25	31.3	.20	53	11.39
562	3.157	7552	388.5	355.7	4.53	-283	-296	28.63	35.2	.20	86	11.42
36.9	3,153	757		756.1	4.68	-286	-3.00	28.52	30.5	.18	<u>82</u> 84	11.49
370	3.157	757. 7583		367•C 76**	4.95	-285		27.5		,24	84	11.50
	3 15.	7590	5 391 0	357.5	4.79	-2F3	-299	28.95	36.2	• 22	24	11+52
37.3	3.155	753.	= 397.5	76.5	4.55		-207	23.87	<u></u>	<u> </u>	<u>85</u>	
	3.159	758		257.5	4.56	-285	-296	28.13	30.2 3 <u>2.5</u>	.22	85 F1	11.54
37 <u>5</u> 376	$\frac{3}{3}$	759	<u>791.5</u> 392.5	<u>].^3r</u> .[J?	4 97	-2P7	-3_4	29.93	31.0	-1.2	82	11.51
	3.137	762	- 702.5	255.5	4.64	-2=6		28.45_				11.65
378	1	17470	426.5	373 • 0	3.68	-129	-97	1623	-1-2	•35	75	12,63
379 -	.137	613	4.1.	<u>415.e</u>		-235. -283		15.27	31+2. 35+9		75 75	12.01
180 381	3.1.5	737.		-61+. 351.C	5.22 5.14	-28 <u>7</u>	-797	29.77	<u> </u>		15	12.03
382	.330	606.		453.5		-266	-179	14.19	35.7	.26	17	12+66
383	-1	798		497.	2.76	2£5.	157	15+84_				_ 12+55
	3 135			350 . 6	5.15	~277	-296	23.69	3(+9	+ 38	76	12.07

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TABLE	13.	(Continued)

	T TIPE	SFCCp			<u>10x</u>	SCAL				FELINA		
N 0 .	• HIN•	ጽዮሥ •	11/5 82	11/5 PC		_U/S TF*P	D.c. 1000					
			711143 2127		DRAIN P576	CAVITY F	DRAIN F	SCEN	PUBLE	TURE STOC	UIS TEMP	TOTAL
85	3.150	75100	385.0	354 • C	4.92	-283	+332	27.31		TRAIN PSIG	F 76	<u>5C</u> FH 12.13
86 87	- <u>3.200</u> -230	<u>75400</u> 5776:	393.5	<u>367.5</u> 445.T	2.30	-283		29.63	23.3		76	13.06
88	.17	£150r	4:3.2	436 .:	3.12	-278	-171 -162	14.98 17.45	35.6 31.6	- 32 - 42	78 78	13.57
89 90	3.150	75400	386.5	257.5	4.79	-282	-299	27.26	32.7	• 46	78	12.85
<u>91</u>	3.150	75450	389.0	763.C 763.5	5,76	-282 -285		29.60	<u> </u>		79	12.89
9 <u>2</u>	3.15	75831	397.5	759 . E	5.25	-290	-296		32.5	•46	81 81	12.86
93 94	.650 3.200	76200	342.0 386.5	211.5 359.C	5.19 5.74	-280	-187 -296	27.79	31.5	•52	83	12+14
95	3.15%	74435	390.5	361.5	5.56	-295	-295	29.91	33.7	<u>+52</u> +54	<u>85</u> 86	13.21
9 <u>6</u> 97	3.150	76501	381.0	<u>*51.5</u> ?51.5	5:10	-282	-299	22.43_	31.4	54	84	12.63
98	-850	766.13	383.5	241.5	5.25	-273 -286	+222 -299	16.13 28.PC	31.4 31.7	.54	83 83	12.60
99 00	3.15	748.32	365.0	255.0	5,29	-285	-3.2	29.51	31.2	*54	85	12.61
01	3.200	75020	383.5 389.C	354 .C	5.64	-285	-299	31.23	31.2	• 5Z		12.73
22_	3.197	767.5	384.5	259 .C	5.48	-281	~297	3. •64 3`•24	31+2 31+2	+ 52	85 85	12.73
13 )4	•533 •152	7610° 96830	391.0 403.0	752.C 372.5	5.40 2.94	-280	-296	23.48	31.3	+ 52	Ç 6 3	12.68
0S	3.133	756()	381.C	353.5	5.54	-285	-103 -327	13.23	<u>31.5</u> 31.2	*52 *56	88 	12.62
36 37	3.23:	7425	385.0	-355.2	4.94	-286	-352	27.86	31.1	*56	94	12.82
38	3.157	739: 1	383.0	361.C	5.50	-285 -283	-3.JC -3C7	3. •58 29 •44	31.5 31.6	*34 *64	25 87	12.83
99	3.272	74525	392.5	367.5	5.72	-285	-327	31.25	31.2	+62		12,86 12,91
19	3.153	77202	390.5	362.0	5.36	-284	-300	29.39	31.1	• 64	86	12.94
12	3.150	76900	385.2	265.5	5.56	-223		29.48 35.59	35.7 31.2	•6Z *36	26 26	12.96
13 14	.10	92700	*****		*******	*****	******	*****	*****	*******	*****	
13-	.285	0985	*****		******	*****	******	*****	******	*******	******	
16	- <u>-</u>	<u>997:</u>	******		******	*****	******	*****	*****	******	******	
17	.093	9733 972:1	*****	*****	*******	*****	******	*****	******	*****	*****	
9	.082	98737	*****	*****	******	*****	049090	005000	******	******	******	And a second second
20-21-		78222	341.	******	20000000	-202-	-242	\$\$\$\$\$\$ \$ \$ \$ \$ \$ \$ \$	26.8	****	*****	
22	.105	583 <u>,</u> 1	*****		******	******	-4-12 ****	****	20.0 \$\$\$\$\$\$	*24 ******	98 20.000	12.29
23	-583- 683-	771.77	******	****		*****	******	*****	*****		******	****
5	-335		368.5	******	1.14	***** -253	-223	****** 2 • 76	****** 25.4	+22+	****** * 92	····
6	•13"	55567	*****	** <b>*</b> ***	*****	*****	*****	*****	346444	***	884444 4 75	11.68 ******
27 28	.100 .080	9420* 8550*	******	******		****** ******	******	******	******	*****	490408 V	
9	.03.	9270	*****	******		\$3\$\$\$\$	4*****	******		******	\$0\$7\$\$\$ \$0\$7\$\$\$	****
	.250	4260	*****	******		*****	*****	*****	*****	******	******	*****
10	. 17 7	527 <u> </u>	*****	******		******	*****	******	******	******	******	
10 11 12	- 160 - 160			****	****	*****	******	*****	\$\$\$\$\$\$\$	*******	******	
0 1 2 3	-100 -040	2752	******								91	11.76
0 1 2 3 4 5	.100 .047 .65" .103	27000 77000 91700	****** 375.0 *****	76P 5	Z.94	-285 *****	-292 *****	18+41	28.3 *****	.20		
10 12 13 14 15	-100 -047 -65n -102 -102	27525 77225 9175 9555	375.0 ******	769.5 ******	2.94 \$3******	\$\$ \$\$\$\$ \$\$\$\$\$\$.	******	******	******	*20 ******* *******		
10 12 13 14 15 16 7	.100 .047 .65" .103	27000 77000 91700	375.0 ******	76P	2.94 \$3******	******	****** ******	<u>******</u> ******	****** ******	4094940 40949404 40949404	****** ****** ******	*****
10 12 3 14 5 6 7 8 9	.100 .040 .65° .100 .100 .177	27020 77220 91720 95227	375.0 ++++++++++++++++++++++++++++++++++++	76P 5 ****** ****** ******	2.94 ******* ******* ******* 2.92	000000 000000 000000 -287 00000	408494 400000 904444 -293 405400	****** ****** ****** ******	***** ****** 22.3 *****	******	- <u>*</u> ?????* <u>*</u>	***** ***** 12,12
D 12 34 5 6 7 8 9 0	.100 .040 .65" .100 .170 .170 .170 .175 .100 .170 .100	77320 917:0 95237 95237 76400 94620 9523	375.0 +++++++ *++++++++ 376.5 ++++++++ ++++++++++++++++++++++++++	76P = 5 ******* ******* ****** ?71 • 5 ****** ******	2.94 ******* ****** 2.92 ******	000000 000000 562320 -287 00000 00000	44444 44444 94444 -293 435440 *4444	\$02344 \$03404 \$204494 \$204494 \$20449 \$20449 \$20449	****** ****** 22.3 ******	+00+0+0 0000000 0000000 0000000 0000000 000000	. 4000000000000000000000000000000000000	400440 040404 12.12 04444 004044
0 12 34 56 7 89 01 2	-100 -047 -65" -102 -107 -107 -107 -107 -107	77220 91720 95027 95027 95227 76400 9522 9522 9522 9522 9522 9522	375.0 000000 \$00000 \$00000 \$00000 \$00000 \$00000 \$00000 \$00000 \$00000	76P = 5 ****** ****** ****** 271 - 5 ****** ****** ****** ****** ******	2.94 444444 444444 444444 2.92 444444 444444 444444 444444 444444	000000 000000 000000 -287 00000	44444 444444 944444 -293 435444 *4444 *4444	\$\$\$\$\$\$\$ \$\$\$\$\$\$ \$\$ \$\$ \$\$ \$\$ \$\$ \$\$	****** ****** 22.3 ******	******* ******* *******	**************************************	12.12 
01234567890123	-100 -847 -65" -102 -177 -177 -177 -175 -175 -175 -175 -177 -177	77320 917:10 95037 95237 95237 76400 95237 76400 95237 95237 95237 9725	375.0 ++++++++++++++++++++++++++++++++++++	76P.5 ******* ******* ****** ****** ******	2,94 444455 444455 444455 44445 44445 44445 44445 44445 44445 44445 44445 44445 44455 44455 44455 44455 44555 44555 4455555 4455555 4455555 4455555 4455555 4455555 4455555 4455555 4455555 44555555 44555555 44555555 44555555 44555555 44555555 44555555 44555555 44555555 44555555 44555555 44555555 44555555 445555555 445555555 4455555555		438434 44484 554444 -293 435500 *24524 *34424 *34424 *34424 *34424	40000000000000000000000000000000000000	• • • • • • • • • • • • • • • • • • •	+ + + + + + + + + + + + + + + + + + +		12,12 12,12 ***** ***** ***** *****
10 12 13 14 15 16 7 18 9 0 1 2 3 4	-100 -047 -65" -102 -107 -107 -107 -107 -107	77335 91733 95237 95237 76467 95237 95237 95237 95237 97257 97257 97257 97257	375.0 000000 \$00000 \$00000 \$00000 \$00000 \$00000 \$00000 \$00000 \$00000	76P = 5 ****** ****** ****** 271 - 5 ****** ****** ****** ****** ******		660060 040000 360000 -287 040000 040000 0400000 0400000000	434434 44444 54444 -293 435444 435544 435544 435544 435554 4355546 435556 435566 4355666 435566666666666666666666666666666666666		••••••• ••••••• 22.3 ••••••• •*•••• **•••• **•••• **•••• **•••• **•••• **•••• **•••• **••••• **•••••	0004040 000400 014 000000 000000 000000 000000 000000		
10 12 12 13 14 15 16 17 18 19 10 1 2 3 4 5 16	-150 -847 -153 -153 -155 -155 -155 -155 -157 -157 -157 -157	2723 c 9173 c 9523 c 9525 c 9555 c	375.0 ******** ******** 376.5 ******** ******** ******** ********	76P . 5 ******* ******* 271 ****** ****** ****** ****** ******			438333 556465 -273 435566 86665 806666 8066666 8066666 8066666 8066666 8066666 8066666 8066666 8066666 8066666 8066666 8066666 80666666 80666666 806666666 806666666 806666666666	545544 433534 433544 433644 449 449 449 449 449 449 449	• • • • • • • • • • • • • • • • • • •	+ + + + + + + + + + + + + + + + + + +		
10 12 13 14 15 16 16 17 18 19 10 1 2 3 3 4 5 16 17	.163 .640 .169 .160 .175 .160 .175 .160 .175 .160 .160 .160 .160 .160 .160 .160 .160	2723.0 9173.0 9523.7 7640.0 5462.0 7640.0 572.0 9523.0 9523.0 9523.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9721.0 9752.0 9721.0 9752.0 977	375.C ******* ******* 376.5 ******* ******* ******** ******* ******	76P . 5 ******* ******* 271.5 ****** ****** ****** ****** ******		040044 040040 -227 +0000 00000 0000 0000 0000 0000 0000 0000 0000 0000	438333 425323 526425 -293 433360 825525 835555 835555 835555 835555 835555 835555 835555 835555 835555 835555 835555 8355555 8355555 8355555 8355555 8355555 8355555 8355555 8355555 8355555 8355555 8355555 8355555 8355555 8355555 8355555 8355555 8355555 83555555 83555555 83555555 835555555 835555555 835555555555	****** ****** ****** ***** ***** ***** ****		**************************************	**************************************	****** 12.12 ****** ****** ****** ****** ****** ****
10 12 13 14 15 16 7 18 9 01 12 23 14 15 16 7 18 9 01 12 23 14 15 16 7 18 9 10 11 15 16 16 17 16 17 17 17 17 17 17 17 17 17 17 17 17 17	-150 -847 -153 -153 -155 -155 -155 -155 -157 -157 -157 -157	2753 77250 9552 9552 9552 76400 9552 952 952 952 952 952 952 952 952 95	375.0 ******** ******** 376.5 ******** ******** ******** ********	76P . 5 ******* ******* 271 ****** ****** ****** ****** ******	2.04 		438333 556465 -273 435566 86665 806666 8066666 8066666 8066666 8066666 8066666 8066666 8066666 8066666 8066666 8066666 8066666 80666666 80666666 806666666 806666666 806666666666	500000 500000 200000 200000 500000 500000 500000 500000 500000 500000 500000 500000 50000000 500000000	****** 28.3 28.3 ****** ***** ***** ***** ***** ***** ****	0000000 0000000 0000000 0000000 000000	**************************************	
10 12 12 12 13 14 15 16 17 18 16 12 13 14 15 16 17 18 19 10	-120 -247 -647 -657 -122 -127	2753 77525 9171-0 9553 9553 9553 9552 9525 9525 9525 9525	375.C 000400 000400 000000	76P . 5 ************************************	2.02 ddares ddares 2.02 ddares 2.92 ddares ddares 2.92 ddares ddares ddares 2.92 ddares ddares ddares ddares 2.92 ddares dd		434399 546542 -293 435650 820542 82054 82054 82054 82054 82054 82054 82054 82054 82054 82054 82054 82054 82054 82054 82054 82054 82054 82054 82056 80056 80056 8		• ***** ****** 28.3 ****** ****** ****** ***** ***** 28.1	******* *******	**************************************	
	.123 .047 .657 .127 .127 .127 .127 .127 .127 .127 .12	2752 77525 9525 76465 9525 9525 9525 9725 6925 6925 6925 9785 6925 9785 6925 9785 6925 9785 6925 9785 6925 9785 9785 9785 9785 9785 9785 9785 978	375.0 000400 000400 000400 0004000 0004000 000400000000	76P.5 ************************************	2.04 dd.a.744 2.97 dd.a.744 2.97 dd.a.744 dd.a.744 dd.a.744 dd.a.744 dd.a.744 dd.a.744 dd.a.744 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.4444 dd.a.44444 dd.a.44444 dd.a.44444 dd.a.44444 dd.a.44444 dd.a.444444 dd.a.444444 dd.a.444444 dd.a.4444444444444444444444444444444444		438890 5764800 -293 438000 8000000		044044           22.3           22.3           4.4444           22.3           4.4444           22.3           4.4444           22.3           4.4444           22.3           4.4444	**************************************	**************************************	
10 12 13 14 5 5 5 7 8 9 0 1 2 3 4 5 6 7 8 9 2 3 4 5 6 7 8 9 2 3	-100 -047 -047 -102 -102 -102 -102 -102 -102 -103 -103 -103 -103 -103 -103 -103 -103	2753 77525 9171-0 9553 9553 9553 9552 9525 9525 9525 9525	375.0 000405 000405 000400 0000000 00000000	76P . 5 ************************************	2.02 dasses 2.02 desses 2.02 desses desses 2.02 desses	0000000 00000000000000000000000000000	dataga d	******     ******     ******     ******		******* *******	**************************************	

TABLE 13. (Continued)

	11 ME	SFEFB RPM	·		LOX	SÇAL				_PELIUN	SFAL	
NO.	MIN+	RPP	U/5 .PR.	375 PC	• P/S PC	LUVS TEMP	0/5 1CHP	LEAPAGE	UZS PR.	DIS PR.	UZS TEM	PJEAKA
			CAVITY	STAL	CRAIN	CAVITY	ORAIN		PURGE	TUPE SIDE	PURGE	TOTAL
			2129	<u>1 c</u>	PSIG	٢	<u> </u>	<u>SCEN</u>	<u> </u>	CRAIN PSIG	F	<u> </u>
55	÷C 50	100000	*****		******	.******	*****	*****	*****	******		******
56	<u>•050</u>	3761	******		******	*****	******	*****	******			******
57 58 1	.CSG 1.5P0	67237	36	746.	2.55	-203	-324	19.45	31.3	• 38	93	7.51
<u>59</u>	.28.	88771	\$\$\$\$\$\$		******		******	0000##	******	******	******	******
6C )	1.557	77500	361.0	346.5	2.47	~296	-354	17.99	31.3	+ 38	92	7.58
	3.520	73750	361.0	271.0			-303 -302	18.75	32.0 31.9	•38 •38	92 95_	7.58
<u>62 4</u> 63	4.421	71856	386	172.5	2.30	-202 *****	*****	1 <u>8.74_</u> *****	******	********	******	******
64	.673	78007	350.5	347.5	2,34	-252	-299	4***	31.0		95	7.56
65	.150	95621	*****		*****	*****	******	*****	*****	****		*****
66	.0E -	9975	*****		******		*****	******	_ ******* ******	\$**********************************	\$\$\$\$\$\$ <b>\$</b> \$\$\$\$\$\$\$	******
67 68	.05. .CSC	9863° 9703°	******		******	*****	******	******	******	3444444		300+0+4
69	-05	E925	*****		******	*****	*****	\$\$\$ <b>\$</b> \$ <b>\$</b>	***3**	******	*****	*****
70	-552	9276*	*****	*****	******	*****	*****	******	*****	******		*****
71	- 55	E29 .	******		******		******	*****	*****	****		*****
72	• ( 52	7162	******		*****		 \$\$\$\$\$\$\$	****** *****	*****	********	 	
73 74	.100	74821	******		******		~~*****	\$\$\$444	******	******	C 2 2 2 2 2 2	
75	. 632	50401	*****	******	******	*****	000000	******	*****	******		******
76	• 3.	6593	*****		******	*****	*****	******	*****	******		******
77	.080	550	*****		*******		\$\$\$\$\$\$ \$\$\$\$\$\$	******	****** ******	******		******
78 79 **	<u></u>	10227	******	-	*******		400000	*****	*****	******		*****
8.		100201	*****		******		*****		*****			******
81	.020	97527	*****		*******		*****	*****	*****	******		*****
	3.320	52833	395.0	771.5	2.56		-294	10,69	25.5	+++++++	96	7.50
83 84	-172 -175	5522 (12)	****** ******		******		******	******	******	*******		******
	3.33	7165	382.5	37.52	2.46		-294	18.68	28.5	.49		7.49
	5.6.22	75907	388.1	•275 .C	Z • 22		-298	19.43	29.5	• 38	94	7.49
	6.653	742.5	385.5	373.5	2.16		-320	17.38	28.8	•38 •38	93 94	7.03
	6.000 6.000	76832	389.0	375.0	2.00		-299	17.35	31.5	.36	92	7.39
93 '	°°°°°	83135	\$\$\$\$\$\$		*******		*****	¢*****	*****	******		*****
	6.00	758.	391.5	577.5	2.14	-295	-299	17.22	32.5	• 36	92	7.61
92	•C.7^	83001	******		******		*****	*****	****	******	<u> </u>	******
93 ( 94	6.000 .100	768LC 846CS	287.5	372.5	2.38		-297 ******	17.98 *****	32.5	•36 ******		/ •01 ++++++
	<del>6303.</del>	76521	392.0	375.0	2.17		-294	17.21	32.1	+36	94	7.63
	6.623	76720	395.0	377.C	2,09	-226	-294	17.20	32.5	• 36	93	7.62
97	• 11 0	E17.	*****		******		******	******	*****	******	47474 90	******
9 <u>8 (</u> 99	<u>6.(13</u> .133	3,61	392.2	375.5	2.35		<u>~3</u> )2	17.25	31.8	*******		7.64
	6.000	76535	395.0	382.0	2.23		+299	18.71	32.0	• 32	90	7.64
	6.1.0	11100	388.5	375.0	2.37		-299	17.96	32.0	• 32	91	7.70
02	•109	<u> </u>	*****		******		*****	*****	*****	******		******
	6.000	7846	394.C	378.5	2.23 *******		-296 *****	17.°4 *****	32•1 *****	• 34 ******	91 ******	7.70
04 05 4	221. <u>223.</u> 6	62335 77255	472.0	785.1	2.14		-294	17.18	<u>32,0</u>		9 <u>p</u> _	
26	.153	82827	*****	*****	*******	******	*****	¢0\$\$\$\$	*****	******		******
	c.oc?_	75507	39	575 • 5	2,22		-30	17.24	72.0 ******	*******	56	7.69
38 20 1	.032	81107	****** 393.C	******	••••••• <u></u> 2.3"		****** -3 <u>,2</u> 7	17.24	31.6	.20	_8.7	7.68
<u>39 1)</u> 16	0.020 .151	<u>75207</u> 80000	******		******			000000	***	******	*****	******
	0.010	75130	391.2	775 + 5	2.43	-227_	-297	_ 17.98	33.4_			7+68
12	.150	83201	******	40¢\$¢¢	******	\$\$\$\$\$\$	*****	\$\$\$\$\$	*****	*******	\$\$\$\$\$\$ 59	******
	១-ខ្មែរី-	76800	395.0	<u> </u>	2.35	-204	-367	12.C3_ ******	33.6	• • • • • • • • • • • • • • • • • • • •	******	
14	.5:3 .130	832.32 76307	074487 274447		******		000000	*****	*****	******	400000	******
16	•13-	776	******	******	******	******	000030	******	*****	******	******	
17	8.002	76250	798.5		2•£1			10.10			95_ ***?**	
18 🕈	****	*****	******	*****	******	* *****		******	444474 44444	*******	******	
19 20	• 273 • 25 ?	71250	******	000000	_0,000,000 + 1 + 1 + 1 + 1 + 1 + 1 + 1 + 1 + 1 + 1	**************************************	******	******	4 <b>*</b> 4 ***	*******	****	******
21	•23	P5601	*****		******	*****	*****	******	*****	******	*****	
	1.9.7	764 . 7	365.0	362.0	3.0'	· -2°3	-324	22.97	25.8	•12	100	9.57
23	.457	715 <u>0</u> *	******	******			******	\$\$\$\$\$\$ \$\$\$\$\$\$	******	********* *******		******
24	.183	47500	******	*****	9009990	040400	*****	·				
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TABLE	13.	(Continued)

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	HIN.	SFCCG RPM			<u>16x</u>					PELIUM	SFAL	
			0/5 P3. CAVIIY PSIC	_J/S_PP_ SFAL SFAL	00010	CAVITY	DRAIN		PURGE	TURB SIDC	PURGE	P_LEAKAGE TOTAL
525	3.252	799: 5	364.5	365.5	2.93	-247	-327	<u>SCFH</u>	28.3	<u> </u>	F 101	<u>sc</u> fh 9.70
526	3.127	77562	385.0	362.0	2.92	-285	-310	25.27	29.0	.36	101	9.70
527 528	3.300	77650	359.5	379.5	3.07	-290	-323	27 33	28.7	* 25	99	9.57
529		569.	0.00044		*****	-289 ******	-302	20.95	29.8	*24	17 <u>C</u> ******	
53 <u>2</u> 531	3-252	_75200_	362.5	175.• S	2.78		-299	19.61	28.4		3~1	
532 532	+152 +6CJ	36767	******		*******	******	******	*****	******	******		
53 3	.11.0	26233	*****		00000000		*****	******	******	*******		******
5 <u>34</u> 5 <u>35</u>	3 202	F792~ 7735	\$4\$\$\$\$		*****	946346	******	****	*****		*****	****
536	-155	875	364.J	77~.5 ******	4.69 \$\$\$\$\$\$	-287 ******	-293 ******	22.22	20.3	1.	110	9.64 *****
537	.123	978.	*****		******	*****	* 20***	*****	444444	******		******
538 539	3.15	75223	382.0	275.6	4.26	-296	-291		2*.8	<u>•12</u>	123	9.60
54 2	3.152	773:**	384.5	371.0	3.94	-292 -294	-296	27.03	28.8	+10 +12	120 99	
541	• 121	927	*****	****	*****	*****	*****	******	*****	******		******
542 543	3.15.	76317	388.5	375.C ?7°.5	4.76	-293	-299	22.69	28.6	•10		9.63
544	.200	295 73	*****	*****	*******	0~2- \$\$\$\$\$#	*****	2:.Y> \$\$\$	29.6 ******	•C8 ******	150	9,66 +++++
595 596	3.200	77000	395.0	381.5	4.34	-226	-293	27.0	22.6	•18		9.63
547	1.37	76525	391.5	783.0	4.14	-226	-293	29.25	28.3	- •56 •22	200	9.66
548_	177	\$1510	*****	*****	******	*****	*****	\$2\$\$\$ <b>4</b> \$	*****	*~~{ \$******	*****	9+37 44444
549 550	3.152	75300	399.C 432.2	381.5 197.5	4.49	-292	-297	27.13	28.8	+ 64	99	
551	3.150	775 53	410.5	385.0	5.04	-289	-298	27.10	28.4		100	9.39
552 553	137	8713° 76932	*****	<b>****</b> **	******	*****	*****	\$\$\$\$¢¢¢	****	00800×		******
554 554	3.253	76900	397.5 392.0	362.5	4.61	-262	-293	27.62	28.6	+*2	99	9.37
555	- 31 -	F6227	*****		****	\$\$\$\$\$\$	*****	21 10	25.8	22.	99 ******	9.36
556 557	2.651	77005	<u></u>		4.15	-285	-294	26+57	28.8		1:0	9.35
55.5	•123	86655	386.2	265.0	5+85 ******	-203 \$\$\$\$\$\$	-297 ******	31.75 ******	31.0 *****	*52 ******	93	1,.01
559	+133	55000	****	*****	******	&¢\$\$#\$	******	600000	202225	4945444		******
563 561	- 213 - 353		******		******	******* ++++***	****** ~~~~~~~~	\$*\$\$ \$*	*****			*****
86 Z	.100	\$5327	*****		******	*****	403884 44444	44444 44444	0990345 4999463	*****		******
563 564	1112	594	*****		******	*****	******	*****	*****	******		******
565	+432	18537 7713	******		******	\$\$\$\$\$\$\$ \$\$\$\$\$\$\$	****** ******	*****	******	*****		******
566	.LET	78.03	***		******	*****	**>**	******	66649 <del>2</del>	4444644 44444		*******
567 568	2.300	2520 T 71630	******		******	******	*****	\$*\$***	695000	******	*****	******
559	.132	F253	385.7	378.5	7.5"	-277 ******	-293 ******	35.67	29.2		20	13.08
572	•13.	2830 *	*****	*****	****	*****	*****	*****	*****	******		\$\$\$\$\$\$\$
571 572	3.12.	7652° 73537	387.0	385.5 361.0	5.74	-295	-296	31.17	29.8	• 65	96	14+13
573->	****	600005	******		*******	-285	- <u>293</u> *****	32.91	3[+0	* 64	96 *****	13.84
574	+152	2630	******		******	******	*****	¢ * * * * * *	****	******	*****	*****
5 <u>75</u> 576	1.653 .135	72000 91407	<u>392.5</u>	344	<u>. 5.57</u> ******	<u>-2*4</u>	- <u>29</u> - \$\$\$\$\$\$	Z5.+29 ******	33 <u></u> ******	****		14+80 *******
77_	. 127	649.	*****	202402	****	******	44444	*****	202400			******
78	2.582	756 . r 857 . r	392.5	386.1	5.87	-289	-299	28.94	33.1	• 76	92	14.95
579	+15: 1+51	776:-	389.0	385.5	######################################	-287	+++++++ -298	3:.78	32.7	<u>+0+4444</u> 	<del>******</del> 96	14.96
581	.102	_ <u>6212_</u>	*****	*****	****	- ******	******	****	******	*****	******	******
82 83	.100 1.350	27602 74300	****** 392.6	****** 3P5."	****** <u>5.34</u>	***** 	####### 	****** 32.32_	****** 29.9	\$\$\$\$\$ \$56	*****	******
589	3.377	76500	368.0	763.5	5+56	-288	~292	3.2.27	29 7	* 90	?>_ 95	<u>13,52</u> 13,52
85		66757	*****		******	******	******	******	*****	*****	******	3000440
586 587	1.102	264:1 741 <u>55</u>	*****	335	****** <u>5.46</u>	****** 	296_	****** ??+23	\$\$\$\$\$\$ 2 <b>\$</b>	◆◆◆◆☆★★☆ • 8월	****** 91	****** 13,59
88	•95)	74623	392.5	365.5	5.10	-289	-290	32.22	29.7	*85	91	13.59
58 <u>9</u>	3:18. 3:100	71950	3. 191	387.5	5.46	- <u>789</u> -785		_ 32+27, _ 77+76	<u>29.2</u> 29.1	• 66	° <u>7</u>	13.63
	2.72	750 JC 785	326.5	750.5 791.45	5.15 4.74	-250	-296	_25.13	20.2	, 24 , 56	92	13.68
92	1.452	77230	392.5	383.0	5.18	-257	-296	32.21	30.0	.86	90	15.09
59 <u>3</u> 594	3.2	F0255 9225	<u>391.C</u>	7F4	5+32 ******	296	-29E *****	20.34	32+6	*******	90	15.46 *******
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									~ , <del></del>		*	

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TABLE 13. (Continued)

TEST TIM				LOX	SEAL						
NO. HIN	• RPH								_PEL108		
		CAVITY		DRAIN	_U/S 1F** CAVIIY	N/S TEM	<u>e leafagi</u>	UVS PR.	DVS PR.	U/S TEMP	LEAKAG
		<u></u>	P*1c	PSIG	F	ORAIN F	SCEH		TUPE SIDE	PURGE	TOTAL
595 3.1 596 2.3					-285	-296		P516 33+5	GRAIN PSIC		SCFH
597 2.5					-??5	-295	29.29	30+5	•60 •62	91 91	15.49
598 6.10					-292 -286	-285	32.55	5.9	.70	90	15. <u>56</u> 5.08
599 1			* *****	* *******	44444	-299	29.68	29.8	• 70	87	14.45
500 1.28 601 .13					-2#6	-296	29.13	29.7	******		*****
602 4.98				******	*****	******	******	******	******	<u>98</u>	<u>14.42</u>
603 3.20	0 7630				-285	~294	27.51	29.2	. 72	90	14.43
604 6.10			389.	5 5.32	~201	-293	29.17	29.8	• 76	90	14.43
606 6.10					*****	******	******	29.2	*******	89	14.47
6.7 2.75					-286	-299	32.78	30.0		******	****** 14.52
638 1.73	J 7030				-286 -286	-297	29.23	29.2	.70	87	14.49
609 .13			* ******	*******	******	++++++	3.13	29.3	•76	87	14.55
610 2.63					-285	-297	28.28	****** 29 <b>.</b> 9	******		******
612 Z.3C				*******	******	*****	*****	*****	+++++++	87 ******	14.55
513 4.75	<u> </u>	r 399.0			-281	-292	29.95	31.0	• 62	86	14.99
614 3.50			385.0	5.32	-289	-289	31.03 29.19,	3.2	•66	•83	15.07
615 2.15 616 .15				5.60	-287	-296	37.60	30.2	•68		15.10
61715		******		*******	******	*****	******	*****	•72 ******	81 ******	15.13
518 2.58	7550'		381.5		****** -255	******	*****	*****	******	******	******
519 2.55	-	396.0	.82	6.34	~284	-293		<u>3[•?</u>	. 70	82	15.07
623 7.65				5.96	-281	-293	32.43	39.7	• 64	83	
622 3.43		**************************************		*******	\$\$ \$\$ \$	******	0+++++	32.1	******	83	15.20
623 .16'		*****	382.5 ******	6.24	-277	-297	31.73	37.0	•74	1 ****** 18	*****
524 2.555			379.5	5.62	******	*****	******	******	******		*****
625 .15			******	*******	******	-293	32.87		.68	81	15.44
526 10.100 627 6.500	72420		375.5	4.55	-267	-295	25.94	******	******	******	
628 - 7.63:	70527		379 C 379 5	5.74	-2*5	-293	32.76	32.6	.62	. 76	15.56
629 7.93			279.5	5.97	-293	-?91	28.77	30.7	.62	76	15+51
630 130	92Cu *	*****		*******	*****	-294 *****	32.99 ******	32.7	.90	72	15.47
631 6.1CT 632 6.CB1	76307 70305	397.5	277.5	6.10	-257	-787	- 31.89 -	****** 3: •Z	******	*****	
633	\$\$\$\$\$\$		377.5	6.62 ******	-226	-236	34.26	37	1.22	74 - 72	14.98
634 3.103	74430		?71.0	<i>⊽≈≈≈≈</i> ≉ 5,54	******	******	******	*****	******	400000	15.26
635 .102	7613-	*****		******	-286	-299	29.44	35.0	1.16	115	23.12
C36 .1C3	22623	*****		******	*****	******	\$\$\$\$\$\$ \$	*******	******	****** *	
637 2.350 638 .100	725UC 8300C	349.5	347.5	5.40	~2<3	-299	29:23		****** 1+44	40000 01 112	
539 3.155	7210	373.5	370.0	******* 5.69	*****	*****		*****	******	112 ++++++++	21.59
540 3.183	737	372.5	367 5	5.55	-296 -292	-303	29.53	31.3	1.22	115	22.22
41 10.	819LC	*****	******		*****		29.48	31.5 ******	1.22	111	22+58
42 3 333	71631	272.5	363.5	5.74	-289	-302	28.51	35.3	******	*****	
44 3+253	73501	.361.5	377•5 375•5	5-68	~293	-303	29.67	31.0	<u>1.36</u> 1.28	113	22.55 21.62
45 3.203	76693	376.0		5.12	-292.	-332	27.12	30.5	1.02	111	21.02
46 1	7973	*****	******	*******	++++++	*****			1 • 14	111	23.97
47' 3.152 48 .060	<u></u>	383.2	377.0	4 . 95	-285	-700	3 <u></u>	31 <u>.8</u>	*404400	*****	****
48 - 080 49 - 100	815. 798:	******	*****			*****			1+10	115	22.90
50 3.150	75335	321.5	379.5	4.76	-293		*****	*****	******		*****
51 3.132	756.55	382.5		4.76	-293 -293	-305 335	26.90	31.8	• 94	157	21.02
52 3.105	74321	382.5	387.5	4.54		-353	26.38	37.8 32.0	<u>5</u> 92	127	21.07
<u>53 3.100</u> 54 112	75227	383.5	372.5	4.54	-299		26.35		• 84 • <u>94</u>	137	21+11
55 3.115	7633-	****** 381.5	****** : 377.5			*****	*****	*****	******	178	21.1 <u>1</u>
56 3.21	743	383.5	379.5	<u>5,29</u> 4,79	-287	-325	27.84	_31.2	.96		1.14
5 <u>7 3. JEC</u>	753:3		367.5	5.27	-285		26.87 27.55	32.2	. 92	429	1.14
56 3.680 59_3.100	74652	385 5	382.0	5.27	-293	-324	27.62	.31.9 31.1			5.54
59_3.1CC_ 52 3.1~;	74700	<u>385 [</u>		5.22		3-1	27.12	_31.1	93		3.59
51 .175	575:-	345.5	177.5 ******	5.6"	-293 ******	-323	24.22	21.3	.94		J•61
52 3.682	7496-	354.5	375.5	5,24	-295		27.6J	****	*******	****** ***	****
3.087	756.	765.0		4.24	-203			31.3 31.3	.94		J 69
54 3.080	75800	383.5	377.0	4.89	-293		27.74	31.3	- •9C •9G	1/6 . ?	
			·		· · · ·	· · · · · · · · · · · · · · · · · · ·				107 2	2.71
			•								
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TABLE 13. (Continued)

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	T TIPE	SFLFD RPP			<u>10x</u>	•				her Ing		·
			CAVIIY	<£41	08414	CAVITY	DRLIN		PURGE	D/S PR. TUPB SIDE	U/S TEMP PURSE	LEAKAG
655	3.133	749 22	751C 383.5	375+5	PSIG 4.92	<u>r</u> ~293	-335	\$CFH 27.36	P\$16 31.3	FRAIN PSIG	<u> </u>	SCFH
	*****	*****	*****		****	*****	44444 44444	\$\$\$\$\$\$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$	*****	\$*\$\$\$\$\$\$		20.7
667	- 102 - 102	0000000 0000000	******		******	******	*****	*****	*****	*****		******
<u>562</u> 669	+ue*	32621	****		*****	\$\$\$\$\$\$ \$\$\$\$\$\$	******	******	******	******		0040041
672	+13"	<u></u>	*****		******	******	******	*****	******	*****		*****
671 672	016. *****	765_7 *****	346.E ******	343.L	3 . C . *******	-284 *****	-290 *****	71.11 900000	32.1 ******	.32 \$\$\$\$\$\$	97 ******	4.5
673	3-153	76500	389.2	379 . C	3.26	-286	~303	20.41	29.7	•42	95	4.3
674	3+150	768 J	388.5	377.5	3.14	-285 -293	-320 -332	17.43	26.1	· • 49 • 49	<u>95</u> 95	4.91
676	3.15-	76921	396.0	389.0	3.28	-293	-363	19.87	30.3	:42	96	6.34
677 678	3.100	772. 769C	396 398.0	785.5	3.19 3.34	-293 -293	~300 ~298	18.31 19.15	31.3	•42	97 97	6.8
679	.: 75	1923	****	******	******	******	*****	00000	*****	******	*****	******
-68 C -68 T	8.820	77610	196.5	312.5	3 91	-284	-299 -332	23.52	29.R 32.9	• 50 • 56	86 82	5.5
682	10.100	77550	394.5	375 . 5	4.25	-285	-299	23.21	29.9	• 56	82	6.3
68 J 68 4	10.27	77257 77355	394.5 392.0	375 • 5 374 • C	4.05 4.78	-287 -287	-303 -323	23.14 23.23	3010	•56 •58	79 #0	6.81 7.2(
685	10.107	7730**	392.0	272.0	4 28	-285	-330	22.64	29.8	• 58	81	7.2
685	16.179	7715-	394 .C	.75.r 377.5	4.16	-297	+303 -313	22.42	30.0	•58	78	7.3
688	3+100	76500	396.5	77+5	3.94	-2*5	-367		31.5	•58 \$45	78	7.3
69 J		75820	395.(	375.1	4.27	-284	-301	2:.19	37.5	+46	78	7.43
- 691		7453	393.5	373.5	4.15	-283	-207	23,54	32.4	•46	78	7.50
692		78520	395.0	275.0	4.16	-285	-298	19.69	32,4	.48	78	7.50
693 694		75130	394 +5	375+5	4=18 4=16	-267 -293	-333 -325	19.60	3C.S 37.4	•50 •50	75 75	7.55
675	3.100	76001	391.5	272.5	4.18	-295	-333	18.52	32.4	.54	75	7.6
-696 -697	<u>3.155</u> 3.155	76721	391.5	772 C 375 S	4 22	-289 -286	-303	18.50	30.4 30.4	•54	75	7.7
678	3+102	753.5	375.5	377.5	47	+285	<u>~352</u>	18.63	50.4	- 54	75	. 7.7
699 735		142.5	*****		*******	0000000 040000	*****	日本市中市市 中市市市市市	*****	*******		******
~70"ĭ	3,150	78322	389.0	279.5	3.76	-283	-25~	27.54	22.5	<u>∍62</u>	93	9.0
702		74722	368.5	279.5	3.73	-287 -287		27.30	29.8	•58	<u></u>	8.9 2+2
704	3.100	75.CC	389.0	397.0	3.74	-2*7	-326	27.22	30.5	.62	91	9.38
735 736		772_*	388.( 395.0	282.5 381.C	3.76	-259 -259	-352 -352	27.01	32.9	+62 +64	95 96	12.01
לכר	3.103	11290	386.5	285.0	3.66	-259	-322	26.43	29.6	.64	96	9.00
708	3.120	77300	389.5	382.C 381.C	3.76	-259 -259		27.23	29.3	•65	96 92	9.00
715	3.152	77320	389.0	382.0	3.22	-259	-317	27.39	28.9	•66	92	9.17
711 712		77755	389.C 388.5	383.0	3.76	-259 -259	7_2- -306-	26.93	28.7	•68 73	93 94	9.2
713	3.05	77523	388.5	352.2	3.97	-259	-325	27,12	23.5	•70 •73	<del>74</del> 94	9.2
714	.1C9 .054	15883 103	*****		*******	******	*****	*****	*****	****		*****
715			******		******	\$\$\$\$\$\$	\$242.00 \$240.00	****** *****	*****	\$\$\$\$\$ <u>\$</u>		******
$-\frac{717}{718}$		<u>681</u>	******		******	*****	******	******	<u>++++++</u>	¢¢¢¢¢¢		******
718 719		F5200 77000	****** 394.C	255.5	4940000 3.16	-292	####### 	****** 24.25	****** 25.8	******* •73	***** 92	******
720	3.20%	742-	395.C	365 . 5	3.46	-293	-327	25.52	35.4	+68	92	125
721			394.5	287.5	3,3"		******	24,94,	3~.4	. 66 *******	94 	_ 11.61 *******
723	3,100	763:20	395.E	263.5	3.34	-257	-3:5	24.36	30.5	+66	95	10.91
724 725			394.£ 792.5	286.5 395.5	3.22	-285 -285		23.95	32.5	68	95 95	11-25
726	3.100	7820-	394 • C	385.2	3 ZZ	-785	3 - 3 - 3	24+37	3:.5	83.	96	11.97
727		78335	<u>394.5</u> 395.0	388.1 788.5	3.47	-295	-322	24,75		66 .64	<u></u>	11.1 10.90
729	3-153	7823	394.5			-202	-305	25+75	29,8	66	93	_11,00
730			394.C 393.C	325.5 784.2	3.34	-20-	-32° -327	24,49	37.5 ?*.5	•68 •68	93 94	11.25
732	3.050	72000	395.1	266.5	3.36	-207	-337	24.49	31.3	+66	94	11.55
~733	3.123	76821	<u>394.C</u> 394.5			2#6	···304 303	24,69 24+37	31.3		95	11.77
124	3*154	1014	27743	. 47 #1	···							

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ORIGINAL PAGE IS OF POOR QUALTRY

## TABLE 13. (Concluded)

11	E 5 T	3411	SFEED			LOX	SEAL .				PELIUN	SEAL	
	NO.	MINi	RPF	U/S PR. CAVITY PSIC	SCAL	<u>P/5_PC+.</u> DR+14 PSIG	UNS TEMP CAVITY	DVS TEND DRAIN	LEAYAGE	U/S PR. PURGE PSIG	DIS PR. TURB SIDC GRAIN PSIG	UJS TEMP Purge F	LEAKAGE TOTAL SEFH
-							777.47		******			•	
		- <b>1</b>	÷			2	1		ŧ		. 1	•	
,	35	3.100	766.55	395 .[	315 794.5	3.42	-295	-323	23.74	31.3	, 74	95	12.12
	36	1+652			195.2	3.27	-285		21.83	35.2		. 95	11.03
	37	3.25	7636		786.1				22.74	29.8	.70	92	11.35
	38	3.100	75153		782.5	3.40			22.39	29.5		92	11.46
	39	3.2103	76763		384	3.42	-296		22.47	29.9	8	93	11.64
-	コア	3.1.0			384.0	3.57	-286	-237	22.92	37 - 7	+65	<b>\$</b> 3	11.74
	41	3,100			385.0	3.55			22.54	31.7	.66	93	11.76
	42	3.253			383.5	3.54			22.27	30.3	. 70	93	11.89
	43	3.100			782.5	• 3.54	-225		22.65	32.5	. 70	94	11.83
	744	2.2'2			382.5	3.46	-285		22.35	32.5		24	
		<del>- ī</del> ð		******		****	*****	*****	*****	*****	*****	****	カキタキキキ
	146	602			382.5	3.57		-217	21.02	30.9	. 73	91	11.95
	47	3.100			786.5	3.67			2 . 98	30.4	.64	90	11.83
-	748	3.207			384.0				21.55	36.9	+74	91,	
	149								2:+29	32+5	+56	90	11.89

# TABLE 14. DATA SUMMARY (SI UNITS)

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EST T	145	57270			LOX	SLAL						·
NO. S	EC.	RPS										
				584L	• P75 PR ORAIN MKNT/So	LAVITY	DIS TEHE	' LEAKAG	EUZS FR Furce	. P/S PR. TURE SIDE	U/S TEMP P6RGE	TOTAL
35	6	*****			34		к	<u>sc=s</u>		MURAIN KNT/		SCHS
36	17						279	.:07312 .:07312	241	<u>0</u>	296_ 294	-00245 -00217
37	2	54 ()					265	.60009	210	10	294	.C0222
38	3	537					252	- <u></u> ::::	Z 10	10	294	.0222
39	<u>4</u> 384	533 422					262	.03711	2:9	9	294	.05222
41	5	617			49		255 256	.03024	2.7	6	294	
42	5	627			105		256	.39237	217	10	293	.20189
43	15	608	1045	985	112		255	.27045	221	15	273	.00184
44 45	10	1187			>139		256	******	214	26	293	.05137
46	9	1178 250			>138			*****	214	23	293	.96127
47	7	450			23		269	.05668 .00011	214 208	3	3 '5	.20227
48	13	601	1_34		32		273	.50514	2:5	10	375	.05217 .30193
49	18		1086		- 75		270	.23748	218	34	324	.33184
50 51	15 15	1453			>138			******	216	34	304	.00127
-52	15	1442 1533			108	302	273	.22377	207	34	315	-UC142
53	126	1312			2 J B		260	.0310 .00132	2 79 2 1 1	37 28	3-4 325	+00231
54	414	1493	1079	993	15		286	.0152	193	37		-00217
_ <u>55</u> 56	56	1475	1048	1048	5	299		*****	279	50		******
57	25 39	1518 1525	******	######################################	******	****			*******			******
58	122	1483		1392	13		*****	*******	208	<u>69</u> 63		******
<u> </u>	40	1423	1362	1048		300	42#**	+C3265	202	53	305	.CO185
-60	57	1433			9		*****	.00260	219	55	326	-20185
61 62	373	1478	1124	1103	11	334	*****	.00281	214	62	37.8	60136
63	12		4444444			*****			******	******		******
64	2.	1522	*******	*******	*******				******			******
65	28		******			****		*******	******			******
56 67	129 378	151C 15CC	1400		19		291	.23339	221	63	335	.00180
69	384	1502	1420	1379		296		+00333	219	62	376	+C0179
69 4		1520	2068	2034	23	290 *****	\$\$\$\$\$	.C0370 .C0384	200	62. 62	305	.C3161
73	384	1500	2:32	2248	23	****	*****	27328		63	*****	-35172
71 72	384	1502	2103	2052	24	****	****	CJ395_	2.08	64	**433	.03171
73	354 384	1493 1498	2117	2249	22 31	*****	*****	.00383	21 <u>4</u>	60	*****	+-0180
74	39	1495	2034	1986	> 1 38	297	297	.33446 .37679	215	<u>61</u> 60	***** 325	.23178
75	30	1475	2234	1955	>138		297	.02669	208	55	-3:4	.00181
76	373	1497	2268	26.26	88	295	****	.63454	221	51	3.23	
77 76	<u>372</u> 15	1517	2375	2013	59	294	****	.0348	219	53	3 24	.00158
79	10	237	1034 1034	1089 1096	4	304 303	223	+00500 +00496	2.52	4	316	.29160
83	13	567	1289	1344	8	323	225	•2.3496 •2.3491	203	10	<u>316</u> 316	.00160 .0184
81	23	812	1069	1124	5	303	230	22485		18	316	.00160
52 83	240	1515	1138	1158	5	3C5	256	.00425	204	64	316	.20170
84	12	1380 867	1027	<u>1069</u> 1055	4		264	.00425	2-7	49	317	.00156
85	43	1455	1041	1392		306	270	.C0491 .C0444	197 207	36 57	318 319	.00156
.86	1 99	_1527	1139	1172	. 6	304	27.5_	00326_	2.73	57	318	+03160 0170.
87	252	15,72	1113	1145		336	290	.uJ316	2:0	64	317	
89	<u>. 372</u>	1497	<u>1131</u>	1165	6	3.76	294.	+C 2315_	275	64		
93	361	***** 1498	******* 1134	******* 115°	\$\$\$\$\$\$ u	****			******			******
91		1513	1103	1103		326	258	.00113	203	66		C01 <u>79</u>
92	327	1478	1.32	1110	3	3,0 <u>0</u>		.C0109	205	66 59		.0C184 0160_
93			*****			****	*****	******	******	*****	*****	
94 95			*******			****	*****	*****	000000 <u>0</u>	*****		** <u>*</u> ** <u>*</u>
96		1342	1048	1052	+++++++ 1	\$\$\$\$ 7 <b>1</b> 1		\$\$\$\$\$\$\$ ,[nr99	*******		*****	
97		1322	1255	1375		<u>301</u> 295	251 272	: 71-4	279	<u> </u>		0000
<u>98</u>	2	*****	******	******	******				\$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$		102 	.50236
99	3	<b>*</b> ***	*****	******	****	*****	*****	******	*****	****	\$\$\$\$\$	
ເວນ	226	1342_ 1317	1400.	1365	<u> </u>	293		_021163.				
			1393	1372	9	293	283	LC0142	257	17		.00250
101	193			1717	17	200	710	30170	3			
	190 <u>54</u> 363	1215	1724	<u> </u>	12	289	268 280	.00179 .00179	203	14	321	00260 00255

ORIGINAL PAGE IS OF POOR QUALITY

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TABLE 14.	(Continued)
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	11 <u>22</u>	SPEED RPS			LOX	SLAL					SEAL	
		hr 5	<u>U/S</u> PR		DRAIN	U/S TEMP	DRAIN			. R/S PR. TURE SIDE	U <u>ZS TCHP</u> Purgc	LEANAGE.
			KNT/SQ	HKNT/SQ	HKNT /Sg	<u>н к</u>	<u> </u>	<u>scrs</u>	KNTZSO	HURAIN KNTZ	<u>s k</u>	SCHS
5	360	1286	2103	2058	14	289	205	+C7259	200	18	323	.CC260
5	362	1292	2103 7189	2068	14	289 294	296 258	.CD208 .Ch222	201 216	18 17-	303 321	.00264 .C3297
17 18	<u></u>	1192	7103	2053	<u> </u>	294	221	.00217	230	20		
9	360	1312	217 3	2252	17	286	28Z	.00231	221	20	299	•00302
Ő.	360	1283	2137	~375	17	286	282 283	.C1231 .C3208	222 238	20 20	299	.CC297
2	361	<u>1290</u> 537	2534	1689 1324	<u> </u>	122	94	.03200		46		.00179
3	24	1192	1717	1573	95	*****	216	.03765	219	15	371	•C0137
4	15	372		*******		****		* ******* *******		******	*****	
5	15			*******		*****		0 2 9 9 9 9 9 9 9 9			¢???* *	
7	45	3B3	979	834	2	197_	143	.03278	212	3		.60132
8	30	1145	1427	1186	34	125	137	53444	213		301	+-0123
9 23	26	485	448	<u>379</u> 627	<u>6</u> 5	<u>114</u> 114	147	-59142 -50175	205	13	320	.03123 .08094
1	13	425	972	796	19	105	151	.60327	200	4	299	.00142
2	- 3	478	******		******	****		*******				00107
3	22 £7	1075	676	<u>586</u> 621	15	110	<u>155</u> 171	.C.3222 .C.3165	213	13	299	.0C123 .CC113
15	13	957	517	303	ų.	113	181	.20189	205	11	297	.00156
6	47	475	986	827	12	107	178	.0236	210	4	297	.0142
7	15	987	1338	******	*****	160	*****	••••	210	14	***** * 295	.00151
9	1L3 162	1125 722	503	427	13	157	191	.00151	171	<b>7</b>	293	.20179
5	67	1102	897	910	10	104	178	CC203	246	14	294	.00184
1.	43	1127	772	696		108	185	.CO2C3 .CO2C3	230	<u>16</u> 15	294	.C0175 .C0160
2 3	123 125	1097 693	- 634 . 476	758 421	1 4	146	193	.00000	212	7	294	.20179
G4	-15	1127			*****	****	*****	******	*****	******	*****	*****
5	32	1120	1165	920	14	137	181	.00245	204	16	294	.03184 .00193
55 57	17 27	1128	758	703	13 10	197 114	135 186	.30227	202 212	16 16	294	-C0208
is	-131	1092	327	745	9	104	189	.20193	2.58	15	291	•C0231
\$9	40	1127	827	731	35	123	163	.00415 *******	219	16	291	.0255
1) 11	111	1135	******* 869	******	******	**** 134	182	.20205	207	17	271	.26245
2	43	1112	827	712		108	185		203	17	291	
13	92	1092	814	738	10	111	189	.07217	201		292	.0260 .0283
14 15	86 149	523 497	889 576	738 717	11 7	111	188 191	.07217 .00142	214 209	6	295	+63278
15	64	517	731	662	8	132	175	.23175	212	6	291	.29293
17	57	530	745	669	10	136	175		257	7	<u>291</u>	<u>+CC293</u>
18 19	302	993 973	1076 924	993	3	161 155	199 215	+00076 +02085	2:7	12 10	293	.CC283 .CC302
50	22	1265	1262	1158	31	****	171	+CC368	210	26	271	.02326
51	6			******		****		******		*****	*****	
52 53	S ~6	1352	******	1076	\$\$\$\$\$\$\$ u	125	*****	•######## •CGCP5	211	28	293	.00316
		1345		1262		167	214	-20271	2 3 5	28	293	.0307
55	144	1383	1172	1103	2	188	228	.00076	205	36	294	.0367
5	245	1427	******	******	****** 3	***** 157	<u>**</u> *** 233	******	27,8	<u> </u>		•C0316
57 58	245			4042400	-	*****		9004000 ·			*****	
59	344	1267	1289	973	1	224	246	+00066	2.29	25	291	.00321
<u>.</u> .	<u>4</u> . u			******		*****		*******			*****	
52 52	5				*******	****		*******			*****	*****
3	63	1367	1586	1475	6	120	222	.LC165	219	30		.62368
4	. #3	<u>. 493</u> 517	<u>1131</u> 1172				21; 199	.20177	202		291 291	0354 
5	13				4044944	****	*****	******	******	******	*****	****
7	96	1472	1517	1527		131		.27368	198	32		
<u> </u>	$-\frac{14}{17}$	<u>1402</u> 1405	<u>1751</u> 1758	<u>1623</u> 1627	35	133	178 177		215	<u> </u>		.::C396 .::J392
59 10	27	1465	1758	1527	29	123 _	179	_+CO340 .	710_	54	291	.20392
ř1 –	Ę4	1397	1731	1626	25	193	185	+17711	2:27	33	291	.C1382
<u>12</u>	141	1425	2537		24	133	174	+00333	210	33		.L2396
13 14	154 50	1397 1395	2313 19 <u>93</u>			104			211			.20396
			<u></u>							·	-	

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	SEC	SPEED 8PS			LOX	<u></u>			~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~	HELIUM	
			CV/11A	SEAL	ΠΩΛΙΝ	CAVITY	LIS TEN DRAIN		FUR CE	- P/S PR. Ture ste	DE PURGE TOTAL
			KN7/59	HENTISQ	HENT/So		<u>*</u>	<u>\$</u>		HURAIN KI	NT/S K SCHS
175	111	1293	2034	1857	29	174	183	107330	<u>200</u>	$\frac{33}{33}$	272 .00382
176	22 35	1395 1392	1965 1965	186P 1834	31 31	15*	178 178	.00354	200	33	292 .0396
178	64	1402	2117	1972	30	luq	179	.53340	213	34	272 .60396
179	71	1393	1965	1513	37	151	157	+22415 +2387	219	32	29C .03297 29C .CJ293
18J 181	92 87	1033 1242	2848 3172	2537	20	101	165	£°368	257	23	201_01293_
182	28	1407	2241	2689	41	103	165	.20439	209	34	291 .C7293 291 .L6288
183	71	1408	2075	1931	29	104	159	.25363 .27363	204	34	291 .10288_ 291 .03302
184 185	24 48	1428 1427	2115	1955	30	124	174	.27373	219	34	201 .63293
186	169	14:2	2368	1965	30	104	178	.0359	214	33 6	291 .C0297 3.6 .C0189
187		327	1169	1255	45	12*	0 <u>3</u> 00000		2°4 \$\$\$\$\$\$	*****	*****
189	6		*****	******	*****	****	*****	******	******		*****
193	5	. – –			******	*****	***** 89	******* •25934	271	******	\$\$\$\$\$ \$75 \$C0198
<u>191</u> 192	11	642	1627	1224	48	195	<u>0</u>	-21052	221	15	3.8 .00198
193	16	800	2137	1586	65	100	93	+11241	228	18	374 .02142
194	19	663	2275 2296	1717 1776	62 43	128	93 94	.01241	233	23 21	334 .00132
195	15	883 1275	*****			*****		*******	*******	******	\$\$255 \$379\$\$\$
197	12	1362	******	****	******	****	*****		******	***** 21	3.9 .L0227
193	15 53	657 1257	2256	1624	63 21	104	94 123	.01255	219 243	29	314 .30955
500	108	1262	2448	2068	19	91	242		248	33	3:3 .60236
201	54	1295	2275	2234	18	99	130	+3G566 +2G585	232	34	<u> </u>
202 203	174	1323 218	2248 1627	1999 1233	19	99	161 89	.03555	212	12	3:0 .03575
204		435	1727	1227	57	172	89	1543	215	23	310 -03574
235	12	527	1796	1255	44	99	<u>84</u> 119	.C1359 .D1209	214	48 >138	310 .09576
206	42 81	1283	2551 2613	1260	32	43	135	.27949	213	>138	300 _00561
2.23	13	1277	2630	1288	117	93	93		2:9	14	3~3 .33670 303 .20657
229	13	1367	2633	1269	125	93	89 95	2954	216	14	374 60639
213	19 1 8 7	1325	2586	1259	43	90	144	+02906 +01.38		12	3.4 .0657
212	63	1292	2556	1271	45	95	138	.21.38	2.4		375 .CC660 315 .CC583
213-214-	15	<u>687</u>	2137	1332	<u>103</u> 102	<u>91</u> 91	<u>89</u> 89	.22533	2:5	10	3 5 .00586
215	13	1143	2669	1360	138	91	93	.23181	278	14	315 12676
216	17	1243	2679	1360	138	91 91	90 91	.03223	214 221	12	3.6 .5:762 325
217	20	1323	2682	1427	103		128	25749			305 .00716
219	19	1357	2630	1439	99	91	89	.23488	217		3:6 .0*669
222	15	1617	2654	1412	103	91 1°5	89 89	.C3752			315 .00723
221	15 14	1328	2913	1429	26	118	83			12	3:4 .6.458
223	11	1343	2723	1431	54	119	83				304 .C0742 103 .C0742
224	14	1243	2758 ******	1931 1931	39 15	113	91 97	*C1575 •50463			333 .60256
225	13 144	1255	2576	1434		117	141		279		323 .C0731 303 .20768
227	189	1267	2517	1434	25	151	149			9 +++***	373 .33768 *****
	<u> *****</u> 13	2452	*****	******	*******	007#¥ 0407#	*****	*****	*****	*****	******
229 233	12		2659	1793	62	1**	<u></u>	. 1519	<u></u>		313 .22695 311 .22692
231	191	12P2	2544 *******	2241	52	*****	345 ****	.21137	22:	4C ********	**** ****
232	$-\frac{13}{13}$	1785	*****	******	******	20.974	****	******	******	******	4430 # #4######
234	13	1397	******	<u>?389</u>	177.	· *** **			221		31200681 313 -00684
235	14	1383	*****	1475		133 Şu		.01055			312 +_0636_
236	<u>185</u> 189				79	1 ,	15.	.01292	2 214	+ 41	211 -5692 312 -50690
_233_	_185	1242	2575	1941	98	102					311
239	186		2565	2317	97 \$\$\$\$\$\$1#	20000	*****		* * * * * * * * *	*******	
243					104	9u	125				113 -CL7C8 313 -CL7C8
242	189	1295	2533	1895		97	125			- 15	312 62710
243 244						97				15	312 .5710
		<u>, , , , , ,</u>	***•*•								
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TABLE 14. (Continued)

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TABLE 14. (Continued)

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<u> </u>			.C.AV1 T.Y	STAL	-DRAIN-	_U/S TEPP CAVITY	DRAIN		FURCE	TURE SIDE	ÊUÊĞE	TOTAL
245	219	1293	2523	2073		H x 97	- *	\$7 <u>85</u> .21753	216	HURAIN KNT. 15	<u>/s x</u> 312	SCHS
246	21	1327	*****	1944	91	120	122	.01703	219	14	312	.00712
247	186	1297 1303	2551	2044	<u>85</u> 83	<u>111</u> 97	<u>1</u> 38 135	01545	214	<u>19</u> 14	<u> </u>	.0715
248 249	186 186	1288	2537	2041	80 80	95	<u>135</u>	+11521	214	14	3-9	.63719
25:	189	1285	2534	2237	73	94	135 130	21497 21729	214	14 14	339	+LG719 CC71 <u>7</u>
251 252	<u>186</u> 189	1265	2482	<u>1982</u> 1868	<u>83</u> 92	<u>97</u> 97	126	-G1826	214	15	311	
253	189	1275_	25.76	1958	81	97	135		212	14	3.9	.03716 .03714
254 255	185- 27	1263	2475	1865 *****	86 *****	36I *****	125 *****	.31643 ******	215	14 ####################################	91'E ****	*****
255	12	1267	****	*****		****	*****		*****			*****
257 258	<u>183</u> 11	<u>1250</u> 1350	2465	1859	78	94 ***	12 <u>3_</u> *****	<u>.</u> 1767_ ********	<u>216</u> . ******	13 *******	375_ *****	########
259	11		*****	4920004	\$46566	\$\$\$\$\$	45744	******		****		******
263	18 185	1352	****** 2554	******* 1923	*******	***** 90	***** 114	.22133	****** 223	*****	##### 324	+++++ +00695
261	185	1297	2692	1941	99	101	113	.12101	222	15	304	+00692
263	125	1298	2648	1975	<u>83</u> 87	94 96	<u>119</u>	01865_ .01861	216	<u>15</u> 14	373	C27C9 CC7C8
264 265	184 187	1275	2648 2658	1937 1931	87 97	9E 97	111	.22191	203	15	3.14	802768
266	184	1270	2630	1944	85 85	99 99	116 117	.01901 .61879	216 218	14 14	304 305	+00699 +00707
267	184	1283	2572	1948	<u>85</u> 96	100	113.		224	15.	315	.0758
269	164	1272	2654	1900	94	101	111	.02114	22 <u>5</u> 224	<u>19</u> 15	325	.00782
273	183	1305	2682 2689	2093 2103	62 55	- 97 98	131	.C1652	224	15	305	
272	125	1293	2645	2136	54	78	125	.01593	225	15	3:6	.CC778
273	185	1262	2574	2018	<u>66</u> 61	120	114	.01974	230	15	376	.20775
275	185	1263	2637	1999	71	97	172	.22229	<u>233</u>	16	374	*65777
276	183	1263	* 2637 2634	1989 1979	72 84	95 97	121 96	12282 102286	233 238	16 15	315 315	+GC775 +CC783
277	185	1285	2648	2058	56	97	116	.01775	232	15	3:5	.63799
279	185	1298	2633	2065	<u>57</u> 95	<u>98</u> 99	<u>116</u> 96	+62604	230		3.5	+65780
253 281	183	1267	2634 2548	1933 1958	75 55	99	105	57101	230	15	335	.02770
282	185	1282	7613	··· 7331	57	96	115	.31927	223	15	3714	+13774
283	183	1283 1275	<u>2623</u> 2589	<u>2037</u> 1955	<u></u>	<u>97</u> 97	· 116	.C1768 .C2151	225	15	<u> </u>	.C0772
285	183	1283	2633	1975	65	96	12)	<u>+C1º36</u>	227	15	3-5	.00718
286 287	185 185	1295	2637 2623	2052	54 53	98 98	125	_C1687 .C1659	225 226	16	3.15 3.76	.00718 . .CC719
288	183	1320	2654	2096	52	99	125	.21638	228	16	3:6	+LC720
289	163	1265	2623	2027	<u>56</u> 74	<u>100</u> 96	122	<u>.[1759</u> .02097	229	<u> </u>	<u>336</u> 374	+22717
291	123	1263	2590	1934	83	97	93	. 12268	236	17	3. 4	2717
292	163 201	1300	2633 2596	2036 1962	70	97 97	105	.02056	232 234	17	324	.CC722 .CC722
294	186	1332	2556		65	98	115	52309	232	17	305	.0724
295	125	1302	2575	2537	59	98	122	.32261 	232	16	30.5	.5:72 <u>7</u>
295	<u>183</u>	<u>1253</u> 1265	<u>2575</u> 2575		79_	99	157	.07233	238	17	3_26 3,16	+10723
299	185	1293	<u>2610</u>	2017	<u>59</u>	96			233	16	374	.L9729
299	63 174_	1272	7592 2551	1982	65 50	96 <u>56</u>	114 122	.02031 3_	232	16 15	304 <u>374</u>	
301	166	1283	2555	1993	50	96	120	•£1723	231	15 16	3.4	
303		<u>1288</u> 1266	<u>2613</u> 2586		<u>54</u> _ 75	<u>96</u> 98	<u> </u>	<u>C1926_</u> 	230	16	354	+
3:4	183	_12° _		1955	65_				23			+53697
305 <u>306</u>	183	1282 1232	2585		61 52	90 90	114 112	-62251 -21895	<u>255</u> 755	1616	3.4	+~*698 +.5699
307	9	597	\$\$\$2¢\$\$	6*00007	******	****	****	*****	******	***	******	******
_308 309	<u>185</u> 183	1292	<u>2562</u> 2572	2072	<u>55</u> 64	<u>97</u> 98	1 <u>?;</u> 173	218 <u>46</u> 	245. 256	<u>15</u> 15	3*3 3*3	
313			?5 <u>{5</u>	199.3				.22196.		16	373	+0.7689
.311	188	1293	2586	2268	46	99	119	71985 71737	219	16 17		+62699
<u>312</u> 313	183	<u>1297</u> 1282	<u>7636</u> 2579		81	123	116	+22379	227	19	334	.L. 583
314	185	1283	2551	1911	95:		· · · · · · · · · · · · · · · · · · ·	.22535				
					_ <b>.</b>			<u>.</u>	····			

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TABLE 14. (Continued)

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	TIPE	SPEED	******	****	L0X - 51	Δι				แกมเบพ	SEAL	
	SEC.	RPS	<u></u>									
-			CAVITY	<u>u/s_pn</u> n 	RAIN I	/S TEMP_D Cavity	∕S TEHP Dr <sub>a</sub> in	• 15 AKAGE	UZS FR. FURCE	E/S PR. Ture Sidê Murain Kni	_ 0/S TEKP Purge /S k	LEANAGE TOTAL - SCMS
	<del></del>		2513	<u>BR1750</u> HK 1917	81753 A 97	X	<u>K</u> 93	_S <u>CHS</u> .32850	231	19	301	.00682
315 315	197	1288 1293	7517	1955	90	97	\$3	.02579	229	19 .	331	.02681
317	185	1303	2544	1958	85	98	95	.02521	228	19	302	.02692
318	185	1293	2548	2613	60	98 101	165	.52175 .55946	219 208	18 3	303	.63761
<u>319</u> 329	<u>18</u> 12	1245	2855	252 P 255 1	11 18	102		.1262	2 28	4	3.13	.10703
321	23	1330	2586	2472	51	101	113	<u>+61554</u>	212	17	373	.50711
322	17	1112	2551	2569	15	101	131	61001	212	5 17	· 303 301	.CO713 .CO715
323	66	1305	2579	2152	<u>31</u> 91	- 99	<u>125</u> 96	+01419	224	18	350	.62727
324 325	184 185	1283 1298	2517 2517	2006	75	irī	105	-115C	2 39	17	3^0	.66732
325	183	1293	2551	1976	77	97	93	.02253	240	17	298 298	.60733
327	185	1295	2551	1979	83 79	97 98	<u>91</u> 93	-55101 -55133	241		275	.62731
328	64	1278 1180	2551 2841	1975 2603	73	101	91	52313	227	16	299	
329 333		172	1813	232	85	190	93	C2 2 2 3		2000000	299	.u3749 .c9652
331	q	1382	2689	2199	141	198	96	.23723	273	29 ******	276	1000s
332				4400494 90 741	19 19	***** 114		******	218	5	376	****
<u>333</u> 3 <u>3</u> 4	30000	> 1667 > 1667	2696	2558	<u> </u>	113		******	234	4		0,7994 <b>8</b> 8
335		>1667	2741	2951	13	104		****	219			000404¥
335	5		2832	2879	19 12	128 114		*******	190	ž	3.4	*****
337		>1667	2775	2827	18	176		*****	158	Z		****
3 39	5	1533	2333	2937	19	126		****	194	<u>}</u>		******
347	8	938	2868	2375	23	101 105		\$\$\$\$\$\$ \$\$\$	210 220	1		*****
341		<u>1533</u> 913	2854	3935	22	136		*****	228	1		****
342 343	191		263	2358	38	98	89	.21477	395	<u>z</u>	321	
344	6	1547	3227	2099	16	113	200	.20606	2C0 258	2 2	335 325	.43414 .22432
343	161	1292	2656	-2433	<u>36</u> 18	<u>99</u> 110	203	.0753	195		3_5	.00417
345 347	5		2889	2996	23	1.78	199	.20976	2:5	5	3-6	.56436
3 48	and the second s		2913	2433	33	122	156	.21115	218	2	3.6	+0.0448 +0.0488
349	5	1472	2958	2079	21	125	<u>185</u> 89	.50010 .u1905	- 228 224	2	333	+52493
352		>1277	2665 2899	235.8 2930	38 11	137	223	.02207	223	2	324	+05458
351 352		>1667	2923	2930	18	110	236	.02207	193	2	3.33	-60426
393	7	>1667	2889	3116	19	117	$\frac{214}{169}$	C_693	210		324	.CC471 .CO493
364			2927 2644	2441 2406	35 36	101 98	91	.61377	210	ż	334	.:G477
<u>355</u> 356				2417	34	97	÷1	.21792	214	2	3_4	+CC490
3 57	18:	1257	2737	2417	34	<u>95</u>		11411	246	<u>2</u>	331 331	.00513
358				2513 2513	33 33	95 97	89 88	+1360	219	ź	302	.02531
359				2482	34	99	91	.01417	214	2	304	-20530
361				2475	34	95	89	.21416	212	2	331 332	+L0535 +_C533
362				2453	31	96 96	86 88	.£1334 .£1352	214 212	ī	302	.L0534
363 364				2472	· 31	97	89	.,1381	212	2	333	.60535
365				2458	+ 32	· 97	89	•C1315	212	2	313	.60536
336	<u>18'</u>	1258	27.56	2458		38	<u>£</u> ?		<u>219</u> 219	<sup>2</sup>	<u>393</u> 333	C0537 .C0538
367				2455	32	99 122	90 91	.C1333 C1351	208	i	1.1	
36 <u>8</u> 369		1263		2455	32	96	55	.01346	209	<u>i</u>	3:1	.6:592
37		7 1262	2717	2482	32	96	87		210	<u>i</u>	302	<u>,1CS43</u> ,543
37				7492 2479	34 - <u>33</u>	97 95	89 53		278 228	2	5.2 <u> </u>	i <del>. 5.2.9</del>
<u>31</u> 2 37				2462	32	<u></u>		.01359	208	5	303	+46543
37	<u>4 18</u>	2 1253	<u> </u>	:479	31	133	91			<u> </u>	323	uCa45 5551
31				2492	321	96 96	26 86			1 G		
$-\frac{31}{37}$	6 <u>18</u> 7 18			2482	32		89			2	321	+L355C
37	88	a 1667	7 2937		25	160		<u>C</u> ^**65	215	ş		+L2568
37	9	8 1359	5 2306	2872	21	125	203	+13721	215	2	297	.07569
38				<u>2489</u> 7420	<u>26</u> 35	<u>9</u> # 10^				3 3	278	+27568
38 35				136		108	156		212	2	205	
38	3	6 133;	3 2361	7247	19	1.5	171	-1.1748	214	3	298	.17569
38	4 18	3 125	2637		35			*0135*		······		
		·			,				*			
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## TABLE 14. (Continued)

EST_1		<u>SPCED</u> RPS	•		Lox,	SLAL				_HELIUN	<u>SFAL</u>	
			U/S PR	<u>. U/S PR</u> SEAL		U/S TEPP		P -LE AKAGE				
					NRAIN HKNT/Sg	CAVITY N w	DR <sup>y</sup> in K	50-5		TURE S		TOTAL
385	189	1252	2654	2441	34	98	89	.01289	210	3	208	SCHS #20572
386	192	1257	2713	2436		96	89	.01798	2 3 0		298	.00616
397	14	962	28 36	3058	16	105	156		245		299	•CC540
388 389	10 187	1358	2779 2665	3006 2465	22 33	125	165 89	.00824 .01297	218 225		299	-UC658
395	186	1257	2582	482		99	29	.1353	225			+UC608
391	189	1262	2682	2486	37	1 100	91	-01425	224		300	.03607
392	189	1263	2741	2544	36	100	91	.01384	224		3.70	+UC669
393 394	<u>39</u> 192	1273	2344	2148	. 36	100	151	.:1279	217		331	.00573
395	189	1245	7592	2430	38	100	91 91	.01436 .01412	232 219	4	323 303	.L0623 .L0596
\$96	1 \$ 9	1275	2627	2420		99,	89	+61340	216		322	
397	<u>3</u> ე	1250	2627	2424	39	104	132	.00761	216	4	3.1	.CO595
398 399	4.6	1277	2644	2355	36	96	89	.51359	. 219		301	·LC595
422	-187	1247	2654	2455	3639	. 97 . 97	89	.C1369 .01474	215	- 4	3:3-3-3-3-3-3-3-3-3-3-3-3-3-3-3-3-3-3-3	.00600
401	192	1276	2682	2468	38	96	89	-01446	215	4	. 31.3	.12651
NJZ	180	1278	2651	2441	38	99	92	.1427	215	4	313	.026.01
403	32	1268	2696	2496	37	120	91	.01128	216	4	3:3	.6598
404 405	188	1613 1263	2779	2555 2437	20 38	131 97	198	.C0624 .C1429	217 215	4 1	324	.03596
726	194-	1237	2654	2455	34		88		219	4	3°23°2	
137	192	1228	2689	2489	3B	97	89	.21443	214	4	303	.02626
100	189	1232	2641	2448	36	98	89	.:1342	218	4	. 3,34	·LÜGű7
129 713	124 129-	1242	2692	2486	39 37	97 98	<u>89</u> 87	+C1387	215	<u> </u>	<u> </u>	100609
*11	192	1268	2665	2455	37	98	89	.01391	212	4	303	.00611 .00612
12	189	1273	2654	2451	38	99	89	.:1443	215	4	3.3	.13612
+13	6	+	** ****			****				******		*****
+14 +15*	5		*****			*****				******		*****
15			******			*****				******		******
117	6		** *****			****	*****	\$*****	******	******		******
+18	5		******			****				*****	*****	*****
119	<u> </u>		*******			*****		******		******		*****
121	zz	1167	2351	231 7	******	93	171	.00468	185	707777FF	<del>₽₽₽₽</del> 3_4	******* .1C580
22	6		*****		******	00000		*******		400000		*****
123	<u>5</u>		*****			****		******				******
124 125	2.7	1252	**************************************	2409	40000 <b>0</b>	***** 98	\$200\$ 131	.2~413	******* 182	¢¢\$\$\$\$\$		******
26			*******	_		44 40 4		*******		******	326	+******
+27	6	1973	******	******	******	****	****	*****	*****	******		*****
125	5		****			*****		******				*****
129	<u>5</u> 15		******			*****		******				******
+31	5		*******			*****		******* ******				****** ******
32	5	1545	******	*****	******	****		******				******
33	2		******			****		*******			*****	*****
34 35	39 6	1253	2526	2541	20	97 **** <del>*</del>	93	******	195	1		.0555
36	6		******			. *****		*******				****
37	ę		******			*****		424444			****	*****
38	93_	1273	2595	255.6	23	0 <u>4</u>				1.		
39	6. 6.		*******			\$\$\$\$\$ \$\$		******* ******				******
41			*******			*****		*******				*****
42	6	1595	******	******	*****	*****	*****	*******	******	******	*****	*****
43	6		** *****			*****		******				*****
44	<u>6</u> 5		******			. \$\$\$\$\$ . \$\$		******* ******			*****	
45	<del>ہ</del>		*******			*****		******			*****	
47	93	1293		2589	20	93				:	34	C 550
49	£	1597	** ****	******	******	<u>**_</u> ***_		*****	000000	******	******	******
49 57	107		*****			5550# 0E		******			****** ¢	*****
51	<u>192</u> 5	<u>1153</u> 16 7	<u>2659</u>	2579	19	95	<u>- 71</u>	2^??3_ *******	4000000	******	3 4	•03549
52	3		******			*****		******				****
	3	1652	******	*****	****	*****		*******			***** *	******
									******	******	*****_*	****
53 54	3	1637	2023204	******	<u>****</u> ****		*.*****	*******	400000			*****

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TABLE 14. (Continued)

ST_T		<u>63322</u>		<u> </u>	LOX	SEAL				HELIUH	STAL	
0.51	EC.	RPS								D/6 00		LEANAGE
_ «»			CAVITY	5[AL		U/5 TEMP CAVITY	DRAIN	. ' LI UN NOU	FUREE	TURI STOE	PURGE	TOTAL
	•				HENT/Sg		×	SCHS		NORAIN KNT	/s¥	SCHS
155	3	1667	******			** **		******	******			******
156	3	s::			******	*****			*****			*******
57	<u>₹</u>		******			****		******* .[?918	*******	3		.cc357
158	95	1117	2482	2386	13	93 *****	÷5	2720000	*****			******
159		1478 1258	2489	2339	17	91	86		216	3	3.5	L0358,
461	211	1228	2489	2558	-15	92	£7	_C2885	221	3	3.6	<u>117358</u>
162	265	1197	2661	Z568	15	93	38	.03884	223	3 \$\$\$\$ <b>\$</b> \$		******
153	5		******			***** 93		******	214	3		
164	43 3	1352	*******	2395	15	72 86.868			****	-		******
165		1662	******	******	****	*****			4694045			*****
457	3	1643	******	****	*****	<u>*\$</u> \$\$\$			******			******
868	3		******			*****			\$\$\$\$\$\$\$\$ \$\$	*****		****
469			*******			04444 04444				449908	\$\$\$\$\$	******
470 471	3		******			4# 44# 4# 44#	4 \$ 7 7 7 4	*****	*****	*****		*****
472		1193	******	******	*****	*****				\$\$\$\$\$\$\$		******
473	3	1247	****	*****	*****	****			*****	\$¢\$\$\$\$\$		*******
474	i		*******			*****				******		****
475 476			******			*****	*****	*****	******	******		******
470	5	92	******	***	*****	****	****	****	******	****		\$\$\$\$\$\$ <del>\$</del>
478	5	17.	*****	******	¢4\$\$\$\$\$	** ***				\$35555 \$250\$\$\$		******
	*****	*****	******	******	*****	*****	*****		*******			******
48) 421	5. 3		*****			*****				*****		*****
482	-27.5	ini				95	92	.32882	197			.03354
483	4	1653	******			****				******		******
484	4		******			**** 95	******				379	
485	200	1193			17	93	<u>, , , , , , , , , , , , , , , , , , , </u>		203		338	+C0353
495 487	363	1237				93	89	.00820	195		357	
488	360	126.		Z586		95	89				37,8 306	
489	360	124				93	89	.00817	214	4+++++		*****
492 .	3	138			*******	94	59				336	
491	362	126. 138				\$\$\$\$ <b>*</b>		434744	******	*******		2227027
493	363	129	2672			56	9_				3*7	. 00359 . 444444
494	- 5	141	-		*****	****	¢¢¢¢¢ 52			* *7*****	3.9	
495	360	127				95 95			and the second s			.00360
495	360 6	127			******	¢\$ \$\$\$		*** * ***	******	******		****
4.78	365	124	5 2524			93	5.8				325	*12361 ******
499	<u>.</u>				******	*****	****			* *****	3:5	and the second s
523	363	127									3,76	
<u>501</u> 522	367	136			******					A 464444		******
503	360	132										*******
5:4	6	136			* *******					1 2		5
505	3.60	128	······································							1		******
536	9	178			• • • • • • • • • • • • • • • • • • •						3,1	60363
507 <u>508</u>	622	125						******	4 4 4 4 4 4	* *****		0.00000
5.9	61 3	125	3 271	261	3 16		8	•19-04	4 23			.07362
510	4	133			* *******			- <u></u>		* ******* ] l	374	95362
511	620	125	269	6 258 * *****				******				*****
512 513	657	128	272	3 76 2	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	92	8	5 \$5785	1 23		331	1 .LG364 . 1019046_
514	32	138	7 *****	* *****	0 #0#0###	20040		* ******	* *****	\$`\$`\$\$\$\$\$\$ ````		· · · · · · · · · · · · · · · · · · ·
515	ė	127	2 000000	* *****	*******		* ***** *****	6 3327909: 9 21229:	* ******	******* *******		
516	3				a andrasa 1 17			9 .C^X5	4 · 21	9 1		
517 518	40	121 2000	2 . 267		* ******		****	* ******	* ******	* *******	****	* ******** * *******
519	16	143	7 802400	* *****	* ******	5. CANA4				* *******		* *******
520	12	118	7 0*****	* ****	* ******		****	* ******	7 950939 4 4544#4	\$ \$\$\$\$\$\$\$ \$ \$\$\$	*****	*******
521			7 ******	* *****	* ********	t ≎¢∋¢i [ 9]		6		ş <u>1</u>		1 6.452
522 523	114	127	3 251	7	¢ \$\$\$\$\$\$\$		* ####	* ******	* *****	* ******		* ******
<u>525</u>	ن <i>ـ</i> ـــ	7	2 10 + 100	* ******	<b>9 \$</b> \$\$\$\$\$\$		****	• <u>*</u> *• <u></u> ***	¢ ¢¢¢¢¢¢	* *******		* ***** <u>*</u>
										•		· · · · ·
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TABLE 14. (Continued)

	<u> 1417</u>				LOX	SEAL					H		
NO.	SEC.	RPS	U/S 29	- HIT Pr									
						<u>- U/S</u> TEMP. CAVITY	DRAIN	4° - L L-AR AU	E UZSTR FURCE	• 075 P TURE	R. UZS SIDE PUR	1611	' LEAKAGE Total
					HKNT/So	н к	ĸ	sc⊷ş	.KAT/SC	HURAIN	KNT/S	5	SCHS
525 526	<u>195</u>	1332 1293			the second s	96	89	.0024		_		311	CC458
527	191	1283				97 94		C0957					.00458
528	1 89	1233					<u>87</u> 88		198		2	***	+02452
529	3				******	****							******
535 531	192	1253				96	89						•uC452
532	36		*******			*****			 \$00000000				*****
\$33	6		******			****			*******				******
534	6		******						******				******
535 536	<u>192</u>	1283	2648			96	*3				l <u></u>	311	
537	7		******			**** ****			******				~^****** *******
538	189	1252		1862	29	96	93						
539	195	13:3	2641	2555		93	91	.51276	199				
54J 541	189	1298	2651 ******	2558		• 92	99	.21198	197				·_C456
542	189	1272	2679			93	89	•C1354	*******				******
543	189	1293	2733	7613	28	94		.C1225	197		-	311	
544 545	12		******			****		*****	******	******	r 4¢	<b>*</b> **	******
545	192	1283	2723	2632	<u>30</u> 29	96	<u>93</u> 93	.01277	197				
547	82	1275	2699	-592	29	93	93 91	.01225	. 195 . 199	( (		311 319	-20456
548	6	1517	******	*******	******	*****	*****		******				******
549 552	189	1272	2751	2644	31	93		.21263	199		·	319	10/141
551	180	1292	2772	2672 2654	28 35	95 96	95	.01279	196				+63443
552			******			70	91	.C1381	198				+00443
553	1 75	1280	2741	2637	32	07	93	.01304	197	 [		310	
554 555	182	1283	27'53	2596	32	97	92	.1279	199			310	+_2432
556	<u>18</u> 156	1283	******	2636	29	*****	***** 92		******	· · · · · · · · · · · · · · · · · · ·			*****
557	368	1282	2661	2517	43	93	90	+11498	214	0	-		+30492
558	7		******			*****			*****				\$\$\$\$\$##\$
559 563	8		******			****			******			~	****
561	21		*******			*****			*******				*****
56Z	- Ē		******			- \$\$4\$\$			*******				****** ******
563	6		******			2020E			******		**:		******
564 565	26 B		*******			*****			******				****
566	5-		050000			*****			******				****** ****
567	5		*******			****	*****	******	******				*****
569 569	138	1193	2654 ******	2610		171							
373	<u> </u>		******			*****			******				****
571	197	1267	2668	7658	40	97	91	-51471	205	****** 4			••• <del>•</del> ••
572	çò	1225	2686	2484	41	97	93	61553	2:07	6		9	.02053
575	<u>****</u>		*******			****			******		***	*** *	****
\$75	6 <sup>°</sup>	1222	2692	2372	. 38	99	***** 94	•C1335	223	-			*****
576			******						220 \$\$\$ <b>\$</b> \$\$\$	ð 40#445¢			+02698
577	11		*******			*****			*****				*****
578	<u>155</u>	136-		2651	41	95	8.9	-C1365	228	5		<u>5-6</u>	
579 523		1425	******	2653	0000000 34	*****	***** 91	*******	******	*******			+10706
581	6		******			*****			******				*****
582	6		******			*****	****	******	******				*****
583 594	81 198	1238 1275	2753	2654	37	97		£1511	2.06	6	3	28	
585	5		2675	2644	******	106			215	6 ******		,ч э* =	•1.7638
586	6		******			****			******				00000 <b>0</b>
597	66	1235	2589	2654	38	156	91	C1393	27 2	6	3	-6	+07641
588 589	<u>57_</u> 191	<u>1243</u> 1198	2726	2659_ 2672	<u>41</u> 38	95	89. 90	C1426	225	6	3		.02641
590	185	1250	7665	7673	36	95	97	.21429 .21310	201	6			.23643 .20646
591	167	13.3	2679	263.2	33	96	\$1	.21233	200	6		`5 <sup>-</sup>	.02647
592	87	1283	26.89	2641	43	96	91	.21523	207	6	3	. 5	.0:712
593 594	192 13	1337	2596 ******	2648	37	96 \$\$\$\$\$	91 *****	+61337	231	6 ******		75	
- / -		<u> </u>				······································					_ ***	<u>77.</u> †	*****
		•								<u>-</u>			
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		,,	·····									,	
		<u>.</u>			· 							·	

"NO, "	SE C .	RPS		·····		<u> </u>		*			SEAL	
			<u> </u>	• <u>u/</u> • Pr	. D/S P8.	UNS TENE	DIS TEN	IP LEAKAG	E U/S F8	• 075 PF	U/S TEP	P LEAKA
			CAVITY CAVITY	SEAL	DRAIN HKNT/So I	CAVITY		5545		TURP 5		TOTAL
595	185	1265	2766	2655	41	97	K 91	+91 <u>4</u> 52	213	HURAIN		SCH 2073
596	1 33	1188	2599	2654	37	97	91				3*6	
597		1278	2717	2648	33			.51442	91	5		
599	365 8	1297	2744	2651 *****	36	96	89		2.25	5	3.74	.0265
623	77	1297	27:3	2647	39	0++ 0+	<u>+++++++++++++++++++++++++++++++++++++</u>	+51373		<del>\$1\$\$\$\$</del> 5		450044
601	3			****		*****		*****				800Je
632	299	1317	2733	2654	34	97	9.	.51298	201	5		.0368
6 33	<u>192</u> 366	1272	<u>2737</u> 2723	2651	37	2			205		375	C[63]
625	500			1001 25*****	******	99	93	.31277	201	5		+0.268
606	366	1297	2723	2651	40	96	89			5	the second data and the second second data and the second data and	404444
_627_	162	1213	2717	7641	37	96		C1382	201			
6C8 609	162	1172	2727	*****	38	36 ****		£1422	202	5		
612	158	1177	2727	2645	34	97	93	0000000 01325	\$\$ <b>\$\$\$</b> \$\$\$	\$#\$\$\$\$\$ 4	The state of the second se	******
511	<u>5</u>			*****		*****		4000000 400000				+0268; ******
512	138	1255	2727	7634	40	99	93		227	4	A CONTRACTOR OF THE OWNER OWNER OF THE OWNER	
$\frac{613}{514}$	285	1295	2751	2648	41	<u>98</u>	95	1464	2:18	5	301	
515	129	1295	2754	2654 2648	37 39	95 96	90 91		208	5		.0713
616	9	1527	*****	*****	*****	****			211	<u>5</u>		 \$2\$69.13
617	<u> </u>			*****	······································	*****	*****	*****				*****
618 619	155	1258 1265	2734 2731	763C	41	97	93		212	5		3711
620	4 5 7	1167	7734	<u> </u>	41	98	93	-C1531 -C1534	212	4		<u></u>
621	5	1597		****	******	****		****				.CC71 ******
622	2	1215	- 2748	2623	43	101		.:1497	207	5		.CJ758
623	-151	1246	2754	******	4244444 19	*****		******		and a second design of the sec		******
625				*******		50	94 *****	+C1457 ******	895	5	3 5	
626	425	1267	2734	2595	31	96	91		212	4	and the second s	
627	39	1205	2751	2613	43		93	.01452	211	4		
628 629	452	1175	2737	2617 2613	41	98	94	-2135P	212	4	298	.CC132
635	ŝ			+++++++	43 ******	4444¢ 46	72	+01555 4444000	212	<u> </u>		
531	366	1272	2741	2523	42	95	95	.61458	278	77	296	2+49444 (27:77
632	365	1172	2748	2603	45	<del>9</del> 6	95	.01517	212	7	ZPS	
<u>634</u>	185	1243	2556	4404000 2555	the second s	44044		******				*****
635	6			2030 4849463	25 *****	)6 \$\$\$\$\$	99 244#3	.21389 2040000	201	8	*****	1091
638	\$		******	****	****	****		******				*******
637 638	121	1175	2412	2396	33	93		C1370	207	10	318	.01119
639	3 129	1262	2575	3551	424244 29	\$\$\$\$\$ 91		*******				******
643	191	1228	2555	:534	38		<u>87</u> 83	-51394 -21391	216	<u>\$</u>	315	.01049 .61.66
641	6			******		****		******				******
642 643	200	1197	2568	2541	43	95	831	.21346	243	9	319	.01069
644	197	1225	2630	2603	<u> </u>	93	<u></u>	<u></u>	219.	<u></u>		<u>. jūzg</u>
645	192	1177	2592	2568	- 35	96	89	-C1280 -C1288	210 212	7	317	-01626
646	***			\$\$\$2490	******	*****	*****	****	\$***** <u>*</u>			******
647 648	129	1372	2641	2613 *******	34	97	83	+C1384	219	8		.01995
649				*******		<u>*****</u>		******				######################################
653	160	1255	263	2617	13				<u></u>			
651	189	1260	2537	7620	32	93	9 9	+21245	212	6	315	.60999
652 653	185	1238	<u>2637</u> 2644	2613	<u>71</u> 32	95 95	<u>67</u>					
_6 <u>54</u>	6_			2013 2027038		22 *****		\$\$\$213. \$\$\$\$\$	214 ******	5. *******		89933 <b>.</b> ******
655	125	1272	2635	26 3	36	96	89	.51314	215	7		
655	197	1222	2544	<u></u>		<u></u>	97					
657 _653	185 _]55	1255 _1243_	2558 26 <u>58</u>	7623 267 (	36 34	97 	89 26	.5130° 	214 714	5	315	22969
659	186	1245	2654	₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩₩ 2613				+C12P3				; :972 ; :973
667		1253	2554_	317_			E7	J. 1332	216			1:975
661 662	6 165		*****	****		****	*****	*****	******	****	84444	*****
- <u>502</u> -663	<u>165</u> 185	1244 1266	<u>2551</u> 7654	2612	<u>. 15</u> 33	97 73	<u></u>	.51277	<u></u>	- 6	<u></u>	-12-976 1 978
664		1263	2644	2599	39	27			216		214	
									the second second			
-6 Alex												
								****	wa	· · · · · · · · · · · · · · · · · · ·		

TABLE 14. (Continued)

TABLE 14. (Continued)

	SEC.		<u> </u>	• U/5 PC	• "/S PP	• U/S TEMP	C/S TF	P LEAKAG	55 U/S- FF	. PZS PR. ' TURE SI	875 TE) DE PURGE	TOTAL
				' SFAL E HKNTZSO	ORAIN HKNTZSA	CÂVIÎY 'H K	DRAIN K	SCHS	KNTZSC	HORAIN KI	ATTS K	SCHS
665		_124 <u>8</u> _	2594	7596	34	93		+C1277	216			.65977
656				******		**** ****	*****	*******	******	******		*****
667	9			******		\$9994	00.000		*******	****		*****
669	5			******		*****		\$\$\$\$\$\$\$ \$				****
675 671	8 36	1367	2339	****** 7344	21	9499# 99	##### 89	1 3996		2	3.9	-C0213
672		*****	******	****		*****		*****	*****			\$0\$0###
673	189	1275	2582	2633	?z 22	95	<u>87</u> 88	+30963	275 183		375	.02235
675	103	1317	2665	5963	19	93	¢ 8	CC823	256	3	3.5	
676	189	1282	3733	2682	• 23	93	89	+50938	259	3	379	.1:299 .03315
_677 _678	- <u>186</u> 189	1283 1282	2730	2661	22		89 90	0262. 02904	216 206		319	0321
679	4	322	*****	*****	*****	****	*****	****				*****
680	573	1283	2734	2637 7592	27	98 96	29 23	.C1112 .C1295	205	3	5 5 371	.0263
681	676	1253	2713	2596	28	97	89	.01095	256	4	301	.20301
653	612	1283	2723	2596	28	96	87	61092	207	<u> </u>	. 299 3^2	
684 685	6.35 6.35	1288	2733 2689	2579 2555	28 28	96 97	27 83	.01087	205	4	300	
686	6:6	1285	2717	2592	29	• 96	87	1259	217	4	299	-L-1345
<u> 687</u>	186	1286	2730	2603	- 29	95	<u>87</u>	230953	210	43	299	
583 589	175	1275	2723	2023 2581	27	5 <u>8</u>	89	.00983	213	3	299	.C3350
692	1 86	1258	2713	2575	29	98	89 89	.20962	210 210	3	299 299	
691	195	1242	2720	2589	29	173	92	10929	210	3	279	.0354
693	186	1268	2723	2535	29	96	87	.17925	210	3	297	
694	156	1272	2713 2699	2579 2569	29 29	94 94	96 97	.00888	216 210	3 4	297 297	
695	186	1267		2565	29	95	87	.00273	259	4	297	.0354
_697	186	1292	2734	2595	29	96	88	-008P3	215	<u> </u>	297	
	186		2748	25.3		4000+ 71		*******				22244
775	3	225		\$\$\$\$\$\$\$		****		*****				*****
701	<u>169</u> <u>196</u>			2617	26	98	09 85	-C13C3 	205		3.7	
723	195		2668		26	96	P 5	-51275	, 210		3:6	LDC435
724	136	1250	2682		26 25	96 111			210 227	ې 4	3.5 3.8	
705		1287	2589	and the second se	25	111	58		226	4	-3.29	00480 .
7.57	1 96	1287	2679		25						3:9	
738 709			2532 2679		25	111	89	-C1285	199	5	326	
-715			2682	262C	26	111	85	. 21293	199	5	3:6	
711			2692		26	111	45 85	.61271	198	<u> </u>	3-7	
712		1292	2679	262 7	25	111	87	.21253	196	5	338	
714	6					****		*******				******
715 716				¢\$\$\$\$\$\$\$ \$\$\$\$\$\$\$\$		70 070 70 070		+4\$\$\$\$\$				******
717		113	******	******	******	****	****	******	\$\$\$\$\$\$	******		\$790 <b>\$</b> 7 <b>\$</b>
718	21 1F6			********* 2655	*******	** ***	<u>*4044</u> P4	**************************************			3 5	+20471
719					24	92	85		210	5	315	
721	165	1283	2720	2658 *******		04 04000	55 *****	*21172	* 213	5 ******	*****	*****
$-\frac{722}{723}$					23	95	25	+01150	210	5		.C3515
724	126	1250	7717	2665		<u>97</u> 97	<u>87</u>		224			
725					22 22	97	87 57	1150	213	5	3 9	510
727	186	1305	2723	:675	• 22		88	.21158	213		3:9	
728							<u>8</u> 4 84		, press		3*7	
729			2717	2655	23	94	•4	.01155	213			
731	186	1285	2713				85				3'8 3'P	-C05#5
732					24	46	۴u	.21165	214	5	328	05 - 55
739		1278					9 <u>7</u>	+21150	219	·	3; 9	567
										•		

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TABLE 14. (Concluded)

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TE	ST_1	1 2 2 1	SPEED			LOX	SCAL				HEC10H	SCAL	
	ò	SEC.	RPS	CVALLA	SEAL BENT/SQ	<b>BRAIN</b>	CAVITY	D/S TFMP DRAIN	LE AKAGE SC~<	FURCE KNT/SQ	PIS PR. TURE SIDE MORAIN 'ENT/	PLRGU	LEANAGE TOTAL SCHS
	;				<u>"</u> ! /	r., 1		•		14 ·	· •	•••	
				a14	ે તે કુડા			.*	1		·1 _	, ,,,,	
7	133	126	1267	2723	2551	24	97	<u>87</u>	.01120	216	<u>}</u>	38	
	36	- 99	1522		265,H	22	97	85	.51039	278	2	1.8	.53521
- 7	137	195	1272	2723	7651	24	93			215	<u> </u>	3^5	<u>LC536</u>
7	38	186	1252	2717	2637	23	94	<b>S</b> 4	-C1057	203	5	306	.00541
3	139	/ 183	1278	2717	2648	24	94	84	.01260	256	5		
	745	126	1257	2720	2648	24	96	140	.01258	213	5	3.7	.22554
	741	196	1267	2723	7654	25	96	85	.01(64	214	5	3~7	<u>.(3535</u>
	142		1267	2717		24	97	86	.016.51	2. 9	5	3.7	+_0559
	743	196	1262	2723	2637	24	97	87	.\$1269	210	5	3 2 9	.00558
	744	132	1282	2723		29	\$7	88	.ciess	210	5	378	
	745		217		******	-		****		******	****	****	****
	746	36	1298		the second statement with the second statement of the	24	93	83	.53992	213	5	3 25	+L2564
	147	196	1125	2744		75		83	.20793	210	4	375	-\$2557
	748	192	1275	2723		26		84	.21022	213	5	305	.00561
	749	162	1275			75			.30958	219	4	3:5	.11557

## TESTER SHAFT DEFLECTION TESTS

A total of 34 shaft deflection, tests (Table 15) were performed with two Bently proximity probes mounted 1.57 rad (90 degreew) apart over the helium seal mating ring to measure deflection. The LOX and helium seals were not installed. The objective was to perform two 6-minute duration tests, one with a slow start of 4188 rad/s/minute (440,000 rpm/minute) to 9425 rad/s (90,000 rpm) and one with a fast start of 4188 rad/s/s (40,000 rpm/sec) to 9425 rad/s (90,000 rpm). Tests 032 and 034 met the objective. The other tests were terminated due to facility problems.

The Bently transducer measurements of the tester shaft deflection at the helium seal location indicated that the maximum peak to peak deflection was 0.000038 m (0.0015 in.) at 4188 rad/s (40,000 rpm) and 0.000025 m (0.0010 in.) at 9425 rad/s (90,000 rpm). The results are summarized in Table 16.

Number	Date, 1975	Duration, seconds	<pre>\$peeds, rad/s      rpm</pre>	Remarks
001	04-29	15	2408 (23,000)	Low-speed cutoff
002		10	2\$13 (24,000)	Low-speed cutoff
003	04-30	10	7538 (72,000	Bearing cavity redline automatic cutoff
004		9	942 (9,000)	Low-speed cutoff
005	-	14	5528 (91,000)	Overspeed cutoff
006		15	2618 (25,000)	Low-speed cutoff
007		15	2827 (27,000)	
008		28	3769 (36,000)	
009		12	2827 (27,000)	
010		36	3874 (37,000)	ł
011		8	9528 (91,000)	Overspeed cutoff
012		9	5528 (91,000)	Overspeed cutoff
013		32		Turbine pressure decay
014	05-02	8		Overspeed cutoff
015		32		
016		1 11		
017		10		
018		S		
019	r I	15		
020		12		
021		15		
022	1	25		
023		20		
024		20		
025	1	21		
026		25	· · ·	
027		25		
028		26		
029		70		
030		120	l t	1 H
031		15	i	Instrumentation malfunction
032		360	9425 (90,000)	Overspeed and low LOX cutoff
033		15	3769 (36,000)	Low-speed cutoff
034		362	9425 (90,000)	Normal operator shutdown

TABLE 15. TESTER SHAFT DEFLECTION TEST SUMMARY

an a		Bently Dis (Spot Face =	
Speed, rad/s rpm	Acceleration Amplitude, g <sub>p</sub>	No. 1 m (in. <sub>p-p</sub> )	No. 2, m (in. <sub>p-p</sub> )
3455 (33,000)	0.6	0.000033 (0.0013)	0.000033 90.0013)
4188 (40,000)	0.8	0.000025 (0.001)	0.000038 (0.0015)
4188 (40,000)	1.0		0.000025 I
4816 (46,000)	0.5		
5130 (49,000)	0.44		
5759 (55,000)	0.40		
6177 (59,000)	0.36		
6806 (65,000)	0.20		
9109 (87,000)	0.40		
9425 Typical (90,000)	0.36		
All response	es are synchronou	15	

## TABLE 16. TESTER SHAFT DEFLECTION TEST RESULTS

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Inspection of the tester hardware revealed that the bearings and slave seals were in good condition. There were indications that the bearing preload spring and both bearing outer races had spun during testing, but no significant damage resulted. The spring compression was increased to ensure sufficient bearing preload to prevent outer race spinning.

#### PRELIMINARY GASEOUS NITROGEN CHECKOUT TESTS

#### NASA Machined Bellows LOX Seal

Build No. 1 Assembly. The tester was assembled with the machined metal bellows LOX seal and the purged double floating ring helium seal (Tables 7 and 8) for the Schedule I preliminary gaseous nitrogen checkout testing at 2618 rad/s (25,2000 rpm) to 9425 rad/s (90,000 rpm).

The installed length of the bellows LOX seal was adjusted to provide an operating spring load of 21.80 N (4.9 pounds) to be consistent with the theoretical hydrodynamic lift force of approximately 22.24 N (5 pound) with gaseous nitrogen at the minimum test speed of 2618 rad/s (25,000 rpm). The bellows free length was 0.00038 m (0.015 in.) over the print tolerance and the spring rate was 35,024 N/m (200 lb/in.) instead of the original design value of 20,409 N/m (117 lb/in.), which required installing the seal at 0.0047 m (0.184 in.) length. The design nominal operating length is 0.0038 m (0.150 in.). The redesigned bellows mean effective diameter is the same as the original design; therefore, the theoretical pressure loads are nearly balanced. The seal is designed for 0.7 pressure balance ratio.

The helium seal turbine side seal ring ID was 0.0000076 m (0.0003 in.) over print tolerance and the dam width was 0.0015 to 0.0017 m (0.060 to 0.065 in.) instead of the print requirement of 0.0011 to 0.0013 m (0.045 to 0.050 in.).

<u>Tests 035-045</u>. A total of 11 tests for a total time of 7.65 minutes, including test point No. 1 was performed. During the first test series, the LOX pressure in the bearing cavity was higher than the 275,790 N/m<sup>2</sup> (40 psi) GN<sub>2</sub> seal purge pressure; consequently, tests 035 through 040 operated with LOX in the seal cavity at 461,949 N/m<sup>2</sup>g (67 psig). GN<sub>2</sub> seal supply pressure was increased to 1,054,898 N/m<sup>2</sup>g (153 psig) for the second series and testing was terminated after three starts at the second test point conditions when the GN<sub>2</sub> leakage past the LOX seal increased significantly, resulting in seal drain cavity pressures above the redline of 137,895 N/m<sup>2</sup>g (10 psig).

<u>Build 1 Disassembly</u>. The posttest 045 seal hardware and inspection data are tabulated in Tables 10 and '11. The helium seal leakage was slightly lower than the assembly values, indicating no wear or deterioration. the LOX seal leakage was 10 to 25 times the build values, indicating deterioration of the sealing surfaces. Photographs of the LOX seal carbon ring and mating ring are shown in Fig. 57 and 58. The LOX seal carbon face was worn 0.00036 to 0.000061 m (0.0014 to 0.0024 in.) and was scored from rubbing contact. The profile traces indicated twisting of either the carbon ring and/or the mate ring. The carbon face was worn more on the inner diameter with a taper of approximately 0.000018 m (0.0007 in.) from the inner edge of the dam to the outer edge of the lift pad. Comparisons of the carbon face surface profiles pretest and posttest are given in Fig. 59 and 60.

<u>Results and Conclusions</u>. The LOX seal carbon wear was apparently caused by the face closing force exceeding the hydrodynamic lift force. The spring load was set to be equal to the theoretical lift force and the pressure load is nearly balanced at the predicted bellows effective diameter. The lift force may be lower due to nonparallel interface film thickness caused by distortion of either the seal ring and/or the mating ring. The bellows effective diameter may be shifting inward due to pressure deflection of the convolution plates, causing a larger pressure closing force.

It was decided to suspend the bellows seal testing and switch to the piston ring seal to evaluate the effect of a smaller closing force by eliminating the bellows effective diameter variable and reducing the spring load. Consideration will be given to measuring the bellows effective diameter at the operating pressure limits.

### Rayleigh Step Piston Ring LOX Seal

<u>Build 2 Assembly</u>. The tester was assembled with a new piston ring LOX seal and the same helium seal that was used on build 1 to continue the schedule I gaseous nitrogen checkout testing. It was agreed to reduce the LOX seal spring load to 13.34 N (3.0 pounds) and to perform the gaseous nitrogen tests at 9425 rad/s (90,000 rpm) to demonstrate satisfactory operation at the nominal design point prior to testing at the low speed limits. The theoretical lift force increases to 66.72 N (15 pounds) at 9425 rad/s (90,000 rpm). It was also agreed to inspect the seals after the first 6-minute test to check for LOX seal wear.

<u>Tests 046-054</u>. A total of nine tests for 10.58 minutes, including one 6.9minute duration test was performed to complete the test objective. The LOX and helium seal performance was satisfactory on all tests. The LOX seal leakage rate of 0.09 m<sup>3</sup>/minute (3.22 scfm) indicates that the seal is operating properly with a film thickness of approximately 0.0000076 to 0.00001 m (0.0003 to 0.0004 in.). The measured helium seal leakage compares to the calculated values for a diametral clearance of 0.000038 (0.0015 in.) as follows:

	Total Leakage m <sup>3</sup> /minute (scfm)
Measured Static	0.11 (3.8)
Measured Dynamic	0.13 (4.6)
Calculated Isentropic Flow	0.15 (5.4)
Calculated NASA QUASC Program	0.10 (3.6)

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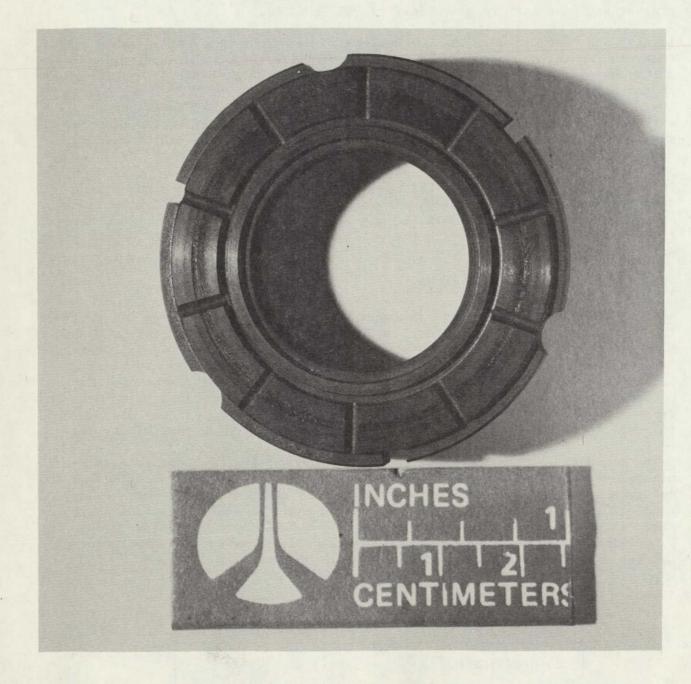
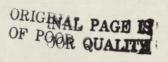


Figure 57. Worn LOX Seal Carbon Ring Posttest 045 (P/N RS009697, S/N 037508, Build 1)



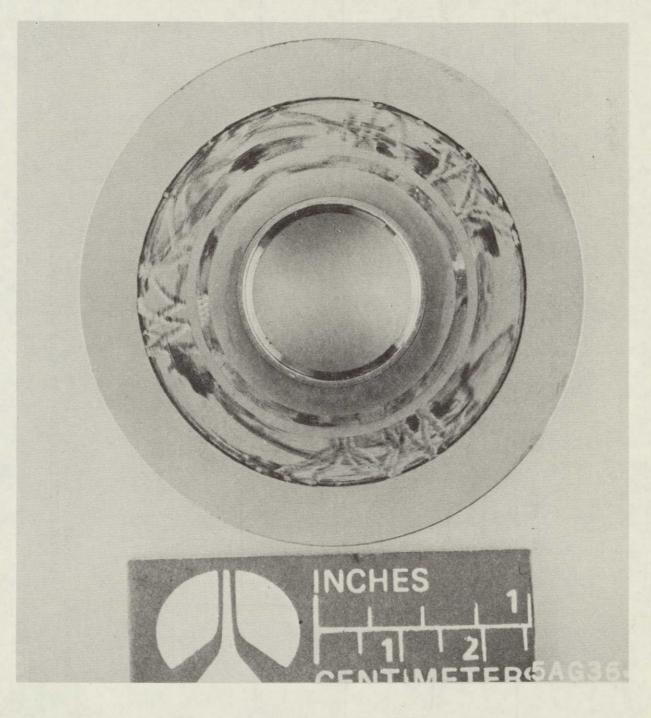


Figure 58. Worn LOX Seal Mate Ring Posttest 045 (P/N RS009698X, S/N 002, Build 1)

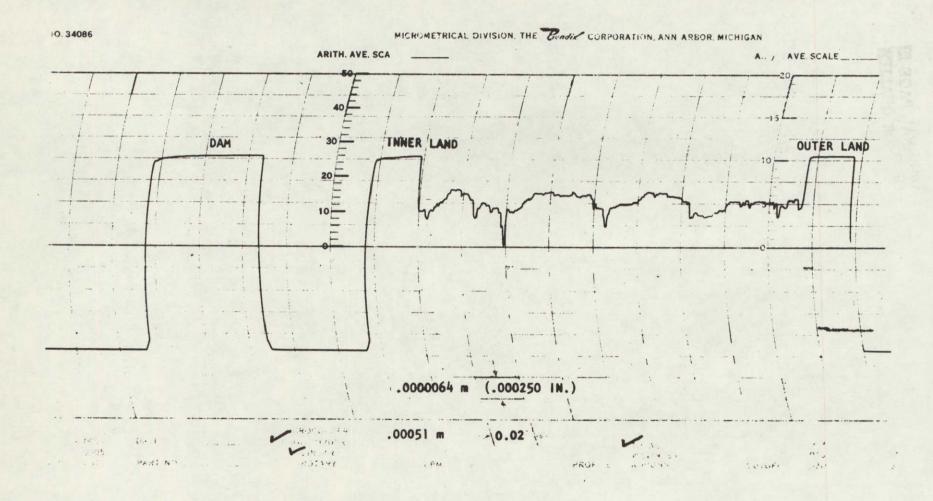


Figure 59. Typical Surface Profile Trace of LOX Seal Carbon Face (RS009697X, S/N 08 New, Pretest 035, Build 1)

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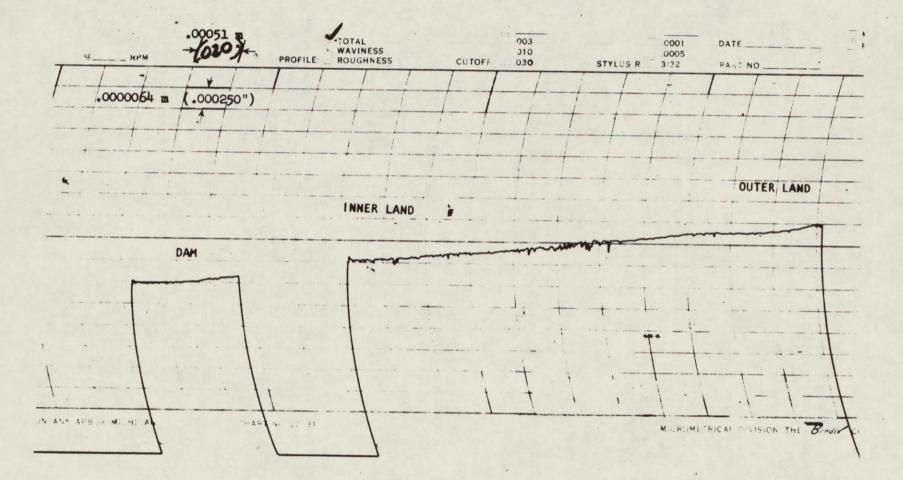
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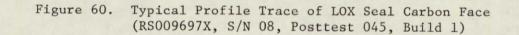
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<u>Build 2 Disassembly</u>. Posttest inspection revealed the tester thrust bearing to be failed. The tester shaft was loose at the seal end in both the radial and axial direction. The shaft had 0.001 m (0.040 in.) axial movement toward the seal end. The bearing balls were flaked and worn with flat spots. The inner race was worn both on the thrust shoulder and down on the race diameter. The bearing preload spring was worn from spinning.

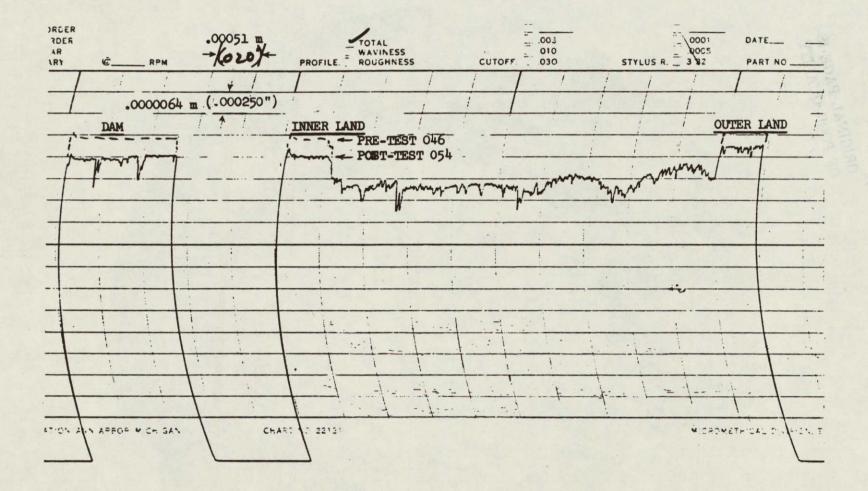
The LOX and helium seals were in good condition, except for slight wear which may have been caused by shaft runout after the bearing failure. The LOX seal face was worn evenly approximately 0.000005 m (0.0002 in.), Fig. 61. The helium seal recess pads were worn 0.000005 to 0.00001 m (0.0002 to 0.0004 in.). The LOX and helium seal posttest static leakage was approximately the same as pretest. The seal inspection data are given in Tables 10 and 11. Photographs of the seal hardware are shown in Fig. 62 and 63.

Investigation of the tester bearing failure indicated that the bearing preload spring had lost preload due to either thermal expansion differentials or wear from spinning. The Belleville spring has a high spring rate; therefore, the preload is greatly affected by a slight change in compression. It was decided to rework the bearing preload arrangement to change from the Belleville spring to a series of 18 compression coil springs located around a plate which loads the bearing outer race (Fig. 64). The coil springs have a low spring rate and will maintain a constant preload of 244,64 N (55 pounds).

It was also decided to rework the tester bearings by installing balls with a 0.000013 m (0.0005 in.) smaller diameter to increase the thrust capacity from approximately 890 N (200 pounds) to 1112 N (250 pounds) for a  $B_{10}$  life of 4 hours. The capacity is approximately 556 N (125 pounds) for a  $B_{10}$  life of 25 hours. The smaller balls increase the bearing contact angle. The calculated bearing thrust load on test 054 was 503 N (113 pounds); therefore, the thrust capacity should be adequate.

<u>Build 3 Assembly</u>. The tester was reassembled using the same Rayleigh step piston ring LOX seal from build 2 with a new carbon seal ring and a new mating ring (Table 7). The same helium seal and mating were reinstalled (Table 8). The tester bearing preload spring was reworked from a Belleville spring to compression coil springs to maintain a constant preload of 245 N (55 pounds). The bearings were reworked by installing smaller balls to increase the contact angle for a larger thrust capacity.

<u>Tests 055-077</u>. A total of 23 tests for 73.8 minutes, including the required ten 6-minute duration tests were performed to complete the remaining  $GN_2$  checkout tests on the Rayleigh step piston ring LOX seal (Table 6). All tests were performed at 9425 ±105 rad/s (90,000 ±1000 rpm). The helium seal pressure was maintained at 206,843 ±13,790 N/m<sup>2</sup>g (30 ±2 psig). The LOX seal gaseous nitrogen pressure was increased in 344,738 N/m<sup>2</sup> (50 psi) increments from 1,034,214 N/m<sup>2</sup>g (150 psig) to 2,084,427 N/m<sup>2</sup>g (300 psig). A total of 9 tests for 45.65 minutes were performed at 2,068,427 N/m<sup>2</sup>g (300 psig). The seal performance was satisfactory.



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Figure 61. Typical Profile Trace of LOX Seal Carbon Face (P/N RS009697X, S/N 05, Piston Ring Seal Pretest 046 and Posttest 054, Build 2)

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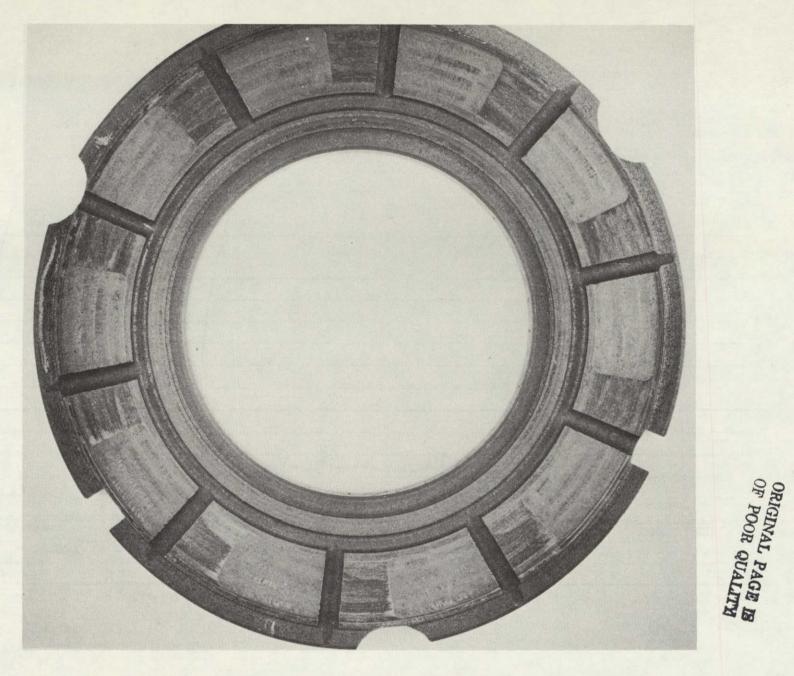


Figure 62. LOX Seal Carbon Ring (P/N RS009697X, S/N 04, Posttest 054, Build 2)

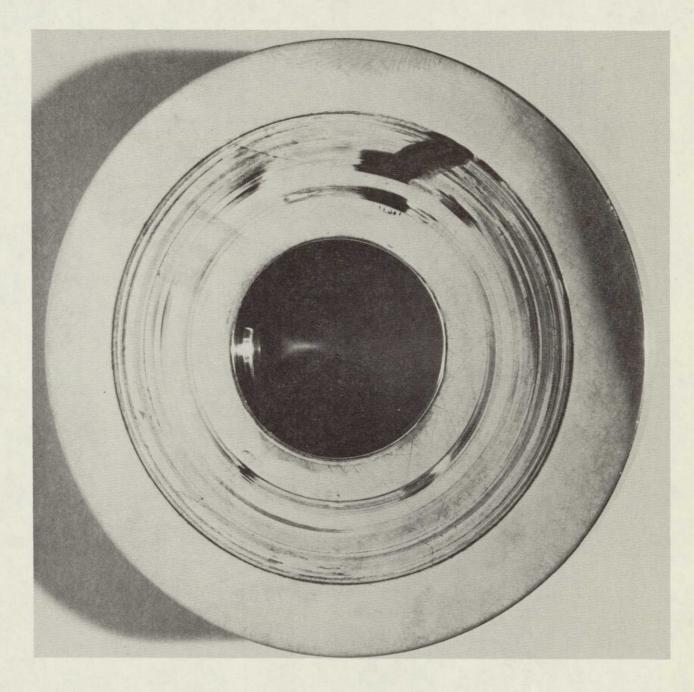


Figure 63. LOX Mating Ring (P/N RS009698X, S/N 01, Posttest 054, Build 2)

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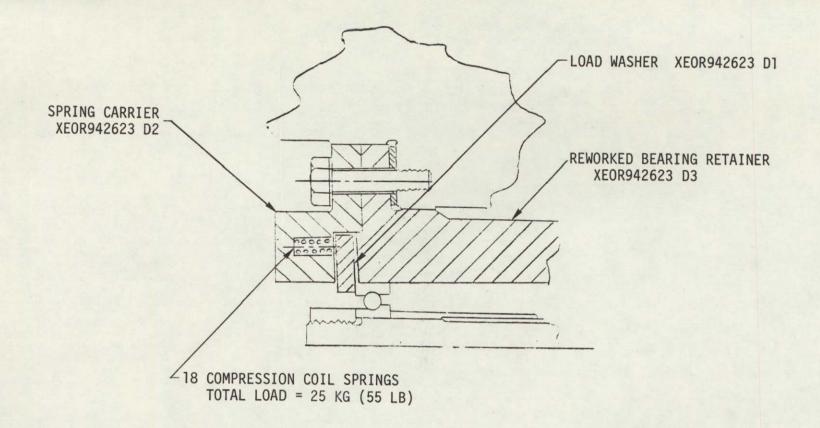


Figure 64. Modified Tester Bearing Preload Spring Arrangement

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The helium seal leakage held steady near the predicted value of 0.10 m<sup>3</sup>/minute (3.6 scfm) throughout the test series. The leakage varied from 0.09 m<sup>3</sup>/minute (3.06 scfm) to 0.13 m<sup>3</sup>/minute (4.65 scfm). Most of the tests varied from 0.10 m<sup>3</sup>/minute (3.6 scfm) to 0.11 m<sup>3</sup>/minute (3.9 scfm). The average leakage was 0.107 m<sup>3</sup>/minute (3.77 scfm). The leakage tended to decrease during the test series.

The LOX seal leakage indicated that the seal was operating properly with a consistent film thickness of 0.00001 to 0.000013 m (0.0004 to 0.0005 in.) as the pressure was increased, except for two tests (074 and 075) where the leakage was excessive. A comparison of the theoretical film thickness for the measured leakage is given below:

Seal Pressure,	Measured Leakage,	Theoretical
N/m <sup>2</sup> g (psig)	m <sup>3</sup> /minute (scfm)	Film Thickness m (in.)
1,034,214	0.156	0.000012
(150)	(5.5	(0.00049)
1,378,951	0.201	0.000012
(200)	(7.1)	(0.00049)
1,723,689	0.221	0.000011
(250)	(7.8)	(0.00044)
2.068,427	0.241	0.000011
(300)	(8.5)	(0.00042)

The excessive LOX seal leakage on tests 074 and 075 was probably caused by either foreign particles or carbon debris between the carbon sealing face and mating ring. The seal appeared to correct itself on the last two tests with the leakage returning to normal.

<u>Build 3 Disassembly</u>. Posttest inspection indicated that the tester bearings were in good condition. The bearings turned smooth and were properly loaded by the coil spring preload arrangement. The bearing end of the tester was not disassembled.

The Rayleigh step LOX seal carbon face was worn evenly across the dam and recess pads 0.00001 m (0.0004 in.), Fig. 65. The recess pads were worn down to a depth of approximately 0.0000025 m (0.0001 in.). The mating ring had a uniform contact pattern with one worn spot. The surface was worn approximately 0.0000076 m (0.0003 in.) in the dam and recess land areas (Fig. 66). The static GN<sub>2</sub> leakage increased by a factor of approximately 10 from pretest to posttest (Table 9 ). The seal was in good condition otherwise, except that the chrome plating on the pilot ring RS009648X was partially flaked off at the pilot. The chrome apparently failed due to a poor bond. Photographs of the LOX seal hardware are shown in Fig. 67 and 68.



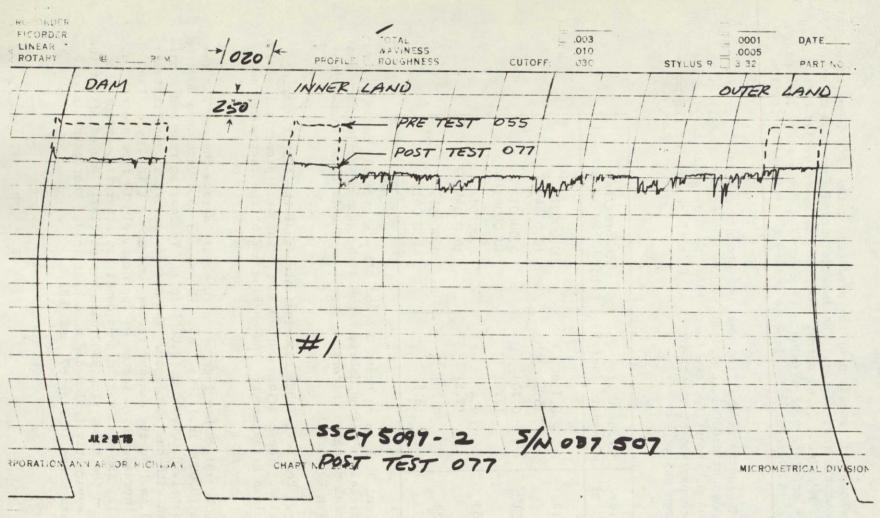
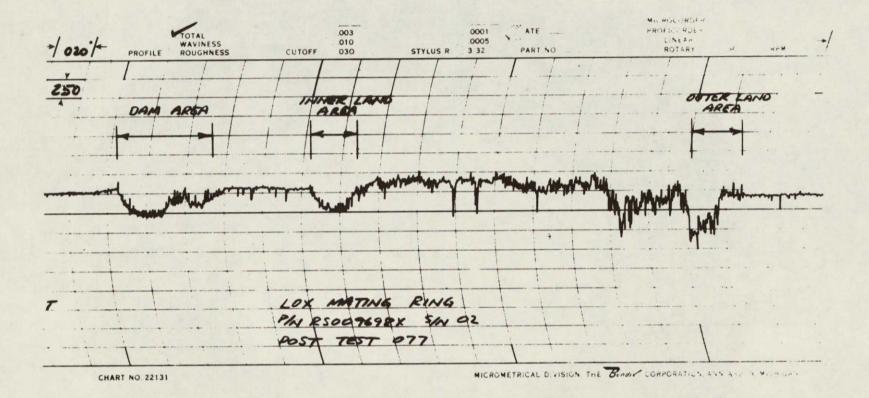


Figure 65. Typical Profile Trace of LOX Seal Carbon Face (P/N RS009697X, S/N 07, Piston Ring Seal Pretest 055 and Posttest 077)

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Figure 66. Profile Trace of LOX Mating Ring Surface At Worn Spot (P/N RS009698X, S/N 02, Posttest 077)

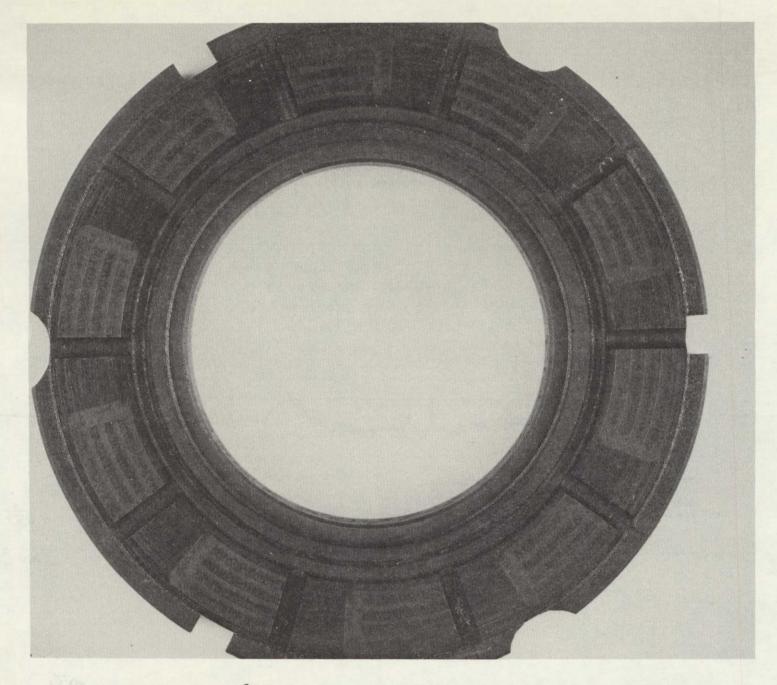


Figure 67. LOX Seal Carbon Ring (P/N RS009697X, S/N 07, Posttest 077)



Figure 68. LOX Mating Ring (P/N RS009698X, S/N 02, Posttest 077)

ORIGINAL PAGE IS OF POOR QUALITY The helium seal was in good condition. There was no measurable wear on either the sealing dam or the recess pads (Table 11). The mating ring was in good condition with only a slight trace of contact. The posttest static leakage was less than the pretest leakage.

<u>Results and Conclusions</u>. Two new carbons and mating rings with the same housing and piston ring were used for 32 starts for 1 hour 24 minutes of  $GN_2$ checkout testing at 9425 rad/s (90,000 rpm) from 1,034,214 N/m<sup>2</sup> to 2,068,427 N/m<sup>2</sup> (150 to 300 psi) LOX seal pressure.

The first assembly was tested for 9 starts and 10.58 minutes at 1,034,214  $N/m^2$  (150 psi) LOX seal pressure. LOX seal carbon eas worn evenly 0.000005 m (0.0002 in.). The mating ring had one worn spot. The tester bearing had failed but the LOX seal was not visibly damaged. LOX seal leakage was 0.09 m<sup>3</sup>/minute (3.22 scfm).

The second build was tested for 23 starts totalling 1 hour, 13.8 minutes, at 9425 rad/s (90,000 rpm) from 1,034,214 to 3,068,427  $\rm N/m^2$  (150 to 300 psi)  $\rm GN_2$  pressure.

LOX seal carbon was worn evenly 0.000013 m (0.0005 in.); the mating ring had one worn spot. The pilot ring had flaked chrome at the pilot.

LOX seal GN<sub>2</sub> leakage plotted versus sealed pressure is shown in Fig. 69. Predicted leakage for GN<sub>2</sub> at 2,068,427  $N/m^2$  (300 psi) and 9425 rad/s (90,000 rpm) could not be determined.

#### Spiral Groove Piston Ring LOX Seal

Build 4 Assembly. The seal tester was reassembled using the alternate spiral groove LOX seal and mating ring manufactured by Crane Packing Co. The same helium seal and mating ring were reinstalled.

Inspection of the LOX seal spiral groove mating ring revealed the ends of the grooves to be closed off on one side with a narrow land due to the cleanup diameter not being concentric. The ring was reworked at Rocketdyne by grinding the lands off and relapping the face.

The initial installation of the LOX seal had excessive static leakage due to the piston ring being 0.00038 m (0.015 in.) loose on the housing. The piston ring was expanded outward against the seal ring instead of being tight at the secondary sealing diameter of the housing. The problem was corrected by making a circumferential wave spring from 0.000076 m (0.003 in.) brass shim stock and inserting the spring around the outside diameter of the piston ring to load it radially inward.

<u>Tests 078-088</u>. The test objective was to perform one 6-minute-duration test on the spiral groove LOX seal with gaseous nitrogen at 1,034,214 N/m<sup>2</sup>g (150 psig) and 9425 rad/s (90,000 rpm) and then inspect the seals.

A total of 11 tests for 20.35 minutes, including the required one 6-minuteduration test was performed to complete the objective (Table 13). The initial tests were terminated due to tester speed control problems. The speed varied



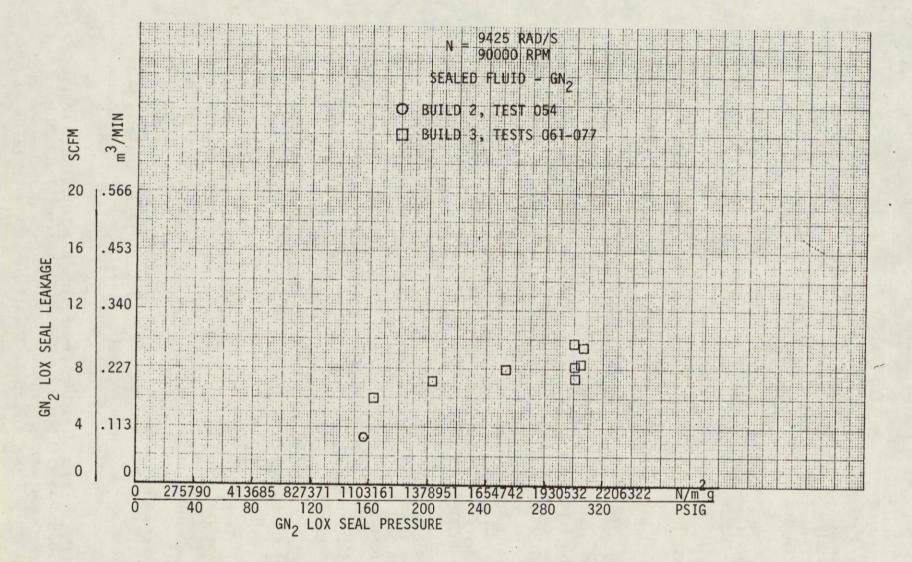


Figure 69. NASA Rayleigh Step Piston Ring LOX Seal GN2 Checkout Tests Leakage

from 1487 rad/s (14,200 rapm) to 9538 rad/s (91,100 rpm). The last three tests were performed at 9425 rad/s (90,000 rpm). The LOX seal gaseous nitrogen pressure was maintained between 1,054, 898  $N/m^2g$  (153 psig) and 1,344,478  $N/m^2g$  (195 psig). The helium seal pressure was 196,501 to 208,222  $N/m^2m$  (28.5 to 30.2 psig). the LOX and helium seal performance was satisfactory.

The LOX seal leakage was steady during the tests and gradually decreased from  $0.300 \text{ m}^3/\text{minute}$  (10.6 scfm) to  $0.190 \text{ m}^3/\text{minute}$  (6.7 scfm) at the completion of testing. The LOX seal drain pressure varied from 3447 to 8273 N/m<sup>2</sup>g (0.5 to 1.2 psig), indicating low leakage rates.

The helium seal leakage continued to hold steady near the predicted value of  $0.102 \text{ m}^3/\text{minute}$  (3.6 scfm). The leakage varied from 0.093 m<sup>3</sup>/minute (3.3 scfm) to 0.100 m<sup>3</sup>/minute (3.9 scfm).

Build 4 Disassembly. Posttest inspection revealed the tester and seals to be in good condition.

The spiral groove LOX seal was in as new condition with no significant rubbing contact and no measurable wear. The carbon face was polished slightly. The mating ring spiral groove surface had very slight traces of light contact when viewed under a microscope. The spiral groove depth was 0.00001 m (0.0004 in.) pretest and posttest. The posttest static leakage was less than pretest (Table 9 ). Typical profile traces of the carbon seal face and the spiral groove mating ring surface are shown in Fig. 70 and 71. The mating ring surface is shown in Fig. 72.

The helium seal was in satisfactory condition. The carbon and mating ring appearance was the same as pretest; however, the carbon recess pads were worn 0.0000025 m (0.0001 in.). The carbon inside diameter was worn 0.000005 to 0.00001 m (0.0002 to 0.0004 in.). The static leakage at 206,825 N/m<sup>2</sup>g (30 psig) decreased from 0.128 m<sup>3</sup>/minute (7800 scim) pretest to 0.089 m<sup>3</sup>/minute (5400 scim) posttest (Table 11). The helium seal hardware conditions is shown in Fig. 73 through 75.

Build 5 Assembly. The tester was reassembled using the same spiral groove LOX seal and floating ring helium seal as build 4 with no rework (Tables 7 and 8).

<u>Tests 089-092</u>. The test objective was to perform eleven 6-minute-duration tests on the spiral groove LOX seal with gaseous nitrogen at 1,034,214 to 2,068,427 N/m<sup>2</sup>g (150 to 300 psig) and 9425 rad/s (90,000 rpm) to complete the schedule I gaseous nitrogen checkout testing.

A total of 4 tests for 13.45 minutes, including one 6-minute-duration test was performed (Table 13). The last test was terminated when the LOX seal drain pressure and bearing cavity pressure redlines were exceeded. Investigation revealed the seals were damaged by a tester bearing failure.

The seal performance was satisfactory prior to the bearing failure. The LOX seal leakage was 0.0623 to 0.0680 m<sup>3</sup>/minute (2.2 to 2.4 scfm) at 1,103,162 N/m<sup>2</sup> (160 psig) gaseous nitrogen pressure. The helium seal leakage was 0.096 to 0.110 m<sup>3</sup>/minute (3.4 to 3.9 scfm) at 206,843 N/m<sup>2</sup>g (30 psig) helium pressure.

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Figure 70. Typical Profile Trace of Spiral Groove LOX Seal Carbon Face (P/N A28-0812-11, S/N 01, Posttest 088)

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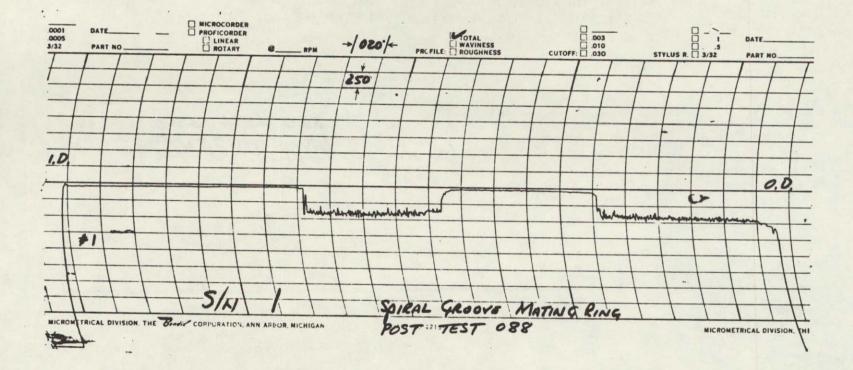


Figure 71. Typical Profile Trace of Spiral Groove LOX Seal Mating Ring (P/N C28-0812-10, S/N 01, Posttest 088)

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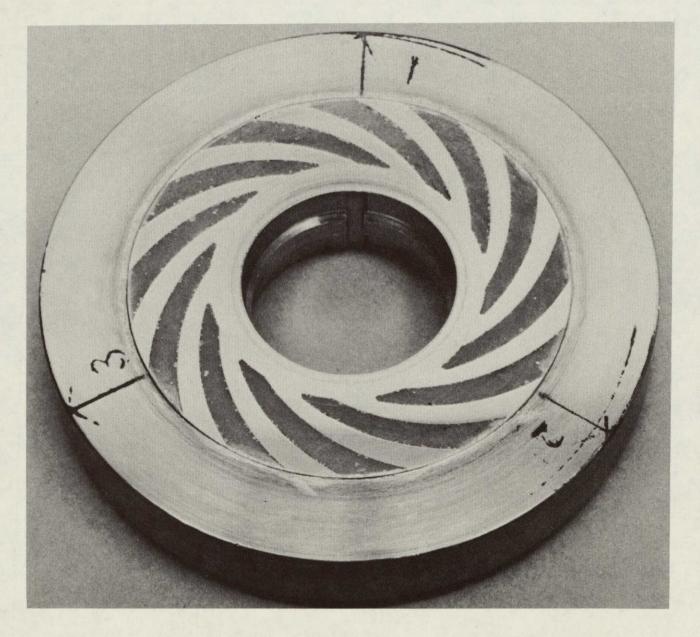


Figure 72. Spiral Groove LOX Seal Mating Ring (P/N C28-0812-10, S/N 01, Posttest 088) OF POOR QUALITY

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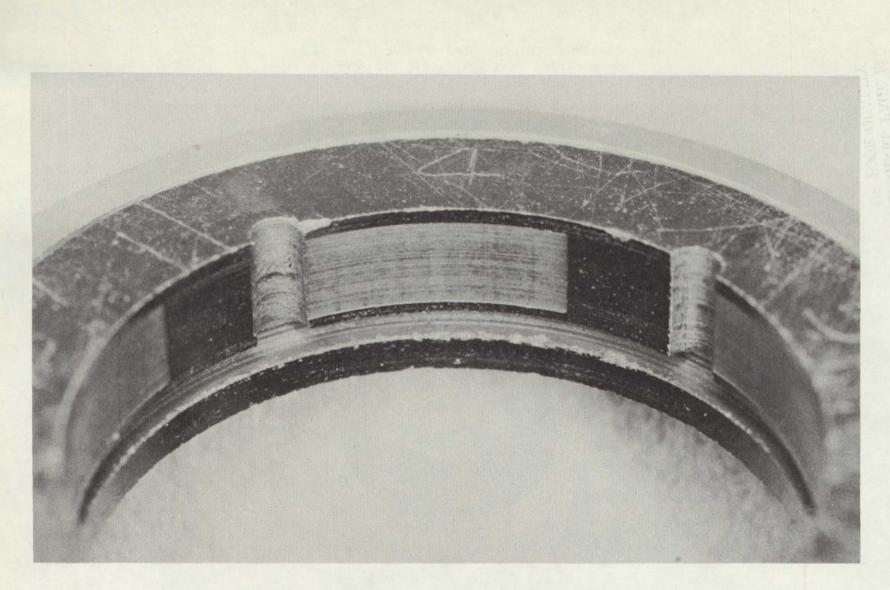


Figure 73. Helium Seal LOX Side Ring (P/N RS009693X, S/N 01, Posttest 088)

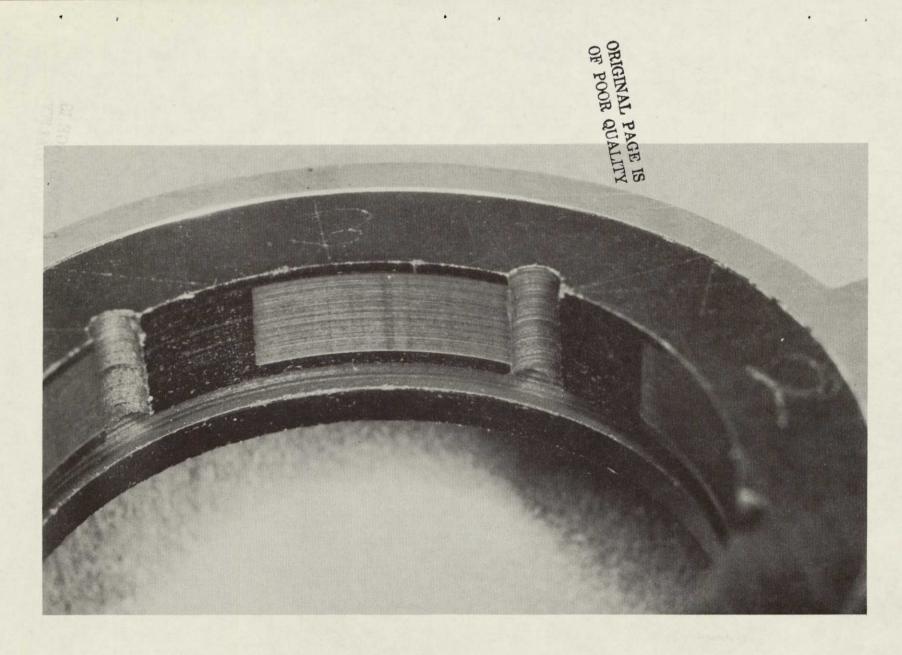
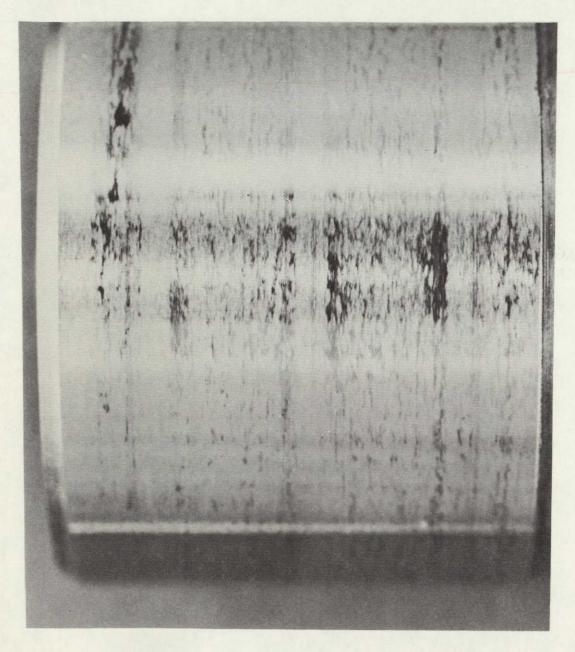


Figure 74. Helium Seal Turbine Side Ring (P/N RS009694X, S/N 01, Posttest 088)



TURBINE SIDE

LOX SIDE

Figure 75. Helium Mating Ring (P/N RS009667X, S/N 02, Posttest 088)

ORIGINAL PAGE IS OF POOR QUALITY <u>Build 5 Disassembly</u>. Posttest inspection revealed the tester bearings to be completely failed. The seals were damaged by excessive shaft radial and axial movements when the bearings failed. The LOX seal carbon was broken and the housing was rubbed by the mating ring when the seal was bottomed out in the axial direction. The surface of the carbon was polished and lightly scored from rubbing contact. The mating ring spiral groove surface was worn slightly on one side from rubbing contact, but was in relatively good condition considering the seal ran in an extreme overload condition. Photographs of the LOX seal hardware are shown in Fig. 76 and 77.

The helium seal carbon rings were chipped and worn due to excessive shaft radial movements. The helium mating ring surface was worn and grooved from the extreme radial load. The seal was in good condition otherwise. There were no structural failures. Photographs of the helium seal hardware are shown in Fig. 78 through 80.

<u>Build 6 Assembly</u>. The tester was reassembled with a new LOX seal carbon ring and spiral groove mating ring, new helium seal and mating ring, new bearings with 0.000013 m (0.0005 in.) smaller balls and the reworked tester hardware.

<u>Tests 093-111</u>. A total of 19 tests for 66.4 minutes, including the required ten 6-minute-duration tests, were performed to complete the remaining GN<sub>2</sub> checkout tests on the spiral groove LOX seal (Table 13 ). All tests were performed at 8062 ±105 rad/s (77,000 ±1000 rpm) and a helium purge pressure of 206,843 ±34,474 N/m<sup>2</sup>g (30 ±5 psig). The LOX seal gaseous nitrogen pressure was increased from 1.034214 to 2,068,427 N/m<sup>2</sup>g (150 to 300 psig) in 344,738 N/m<sup>2</sup> (50 psi) increments. A total of seven tests for 42.0 minutes were performed at 2,068,427 N/m<sup>2</sup>g (300 psig).

Helium seal leakage increased from 0.142 to 0.187 m<sup>3</sup>/minute (5.0 to 6.6 scfm) during the test series. The average leakage was 0.164 m<sup>3</sup>/minute (5.78 scfm). LOX seal leakage repeated previous test values of 0.062 m<sup>3</sup>/minute (2.2 scfm) at 1,034,214 N/m<sup>2</sup>g (150 psig). Leakage increased with seal pressure as shown below:

Sealed Pressure,	Average GN2 Leakage,	Average Drain Pressure,
N/m <sup>2</sup> g (psig)	m <sup>3</sup> /minute (scfm)	N/m <sup>3</sup> g (psig)
1,034,214	0.062	4137
(150	(2.2)	(0.6)
1,378,951	0.085	8963
(200)	(3.0)	(1.3)
1,723,689	0.108	11,721
(250)	(3.8)	(1.7)
2,068,427	0.130	15,168
(300)	(4.6)	(2.2)

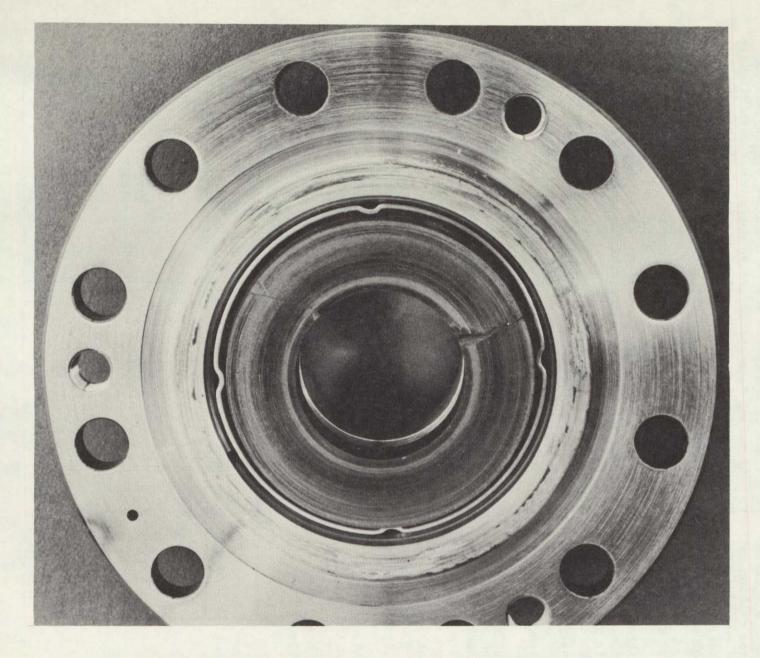


Figure 76. Spiral Groove LOX Seal (P/N RS009695X, S/N 01, Damaged by Bearing Failure Posttest 092)

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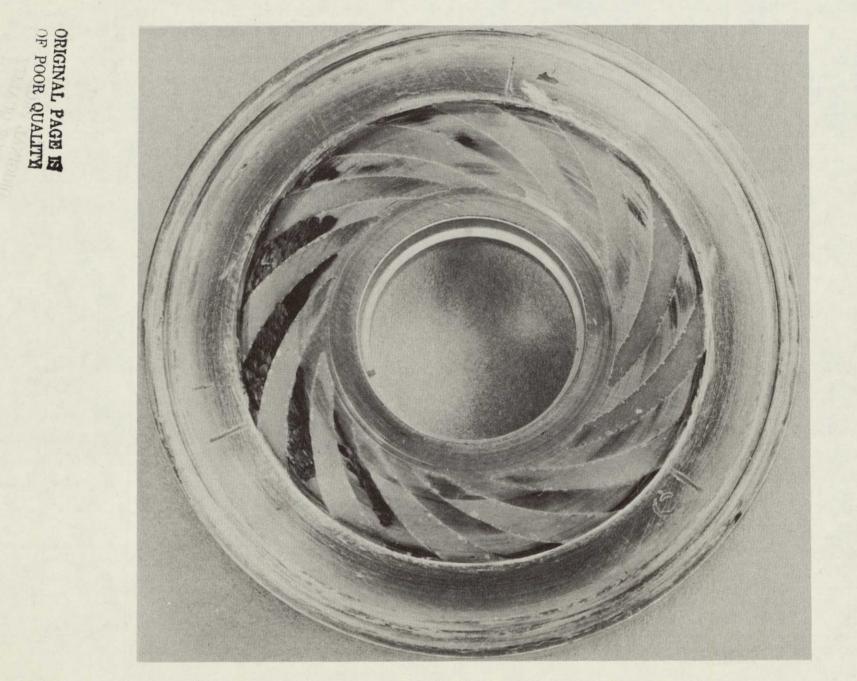


Figure 77. Spiral Groove LOX Seal Mating Ring (P/N C28-0812-10, S/N 01, Damaged by Bearing Failure Posttest 092)



Figure 78. Helium Seal LOX Side Ring (P/N RS0096393X, S/N 01, Damaged by Bearing Failure Posttest 092)

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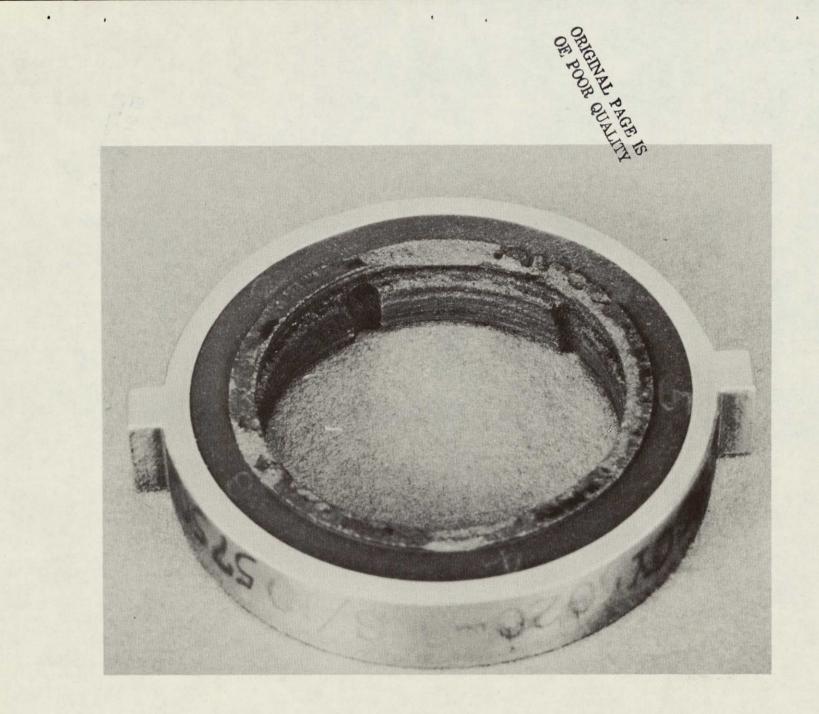


Figure 79. Helium Seal Turbine Side Ring (P/N RS009694X, S/N Ol, Damaged by Bearing Failure Posttest 092)



TURBINE SIDE

LOX SIDE

Figure 80. Helium Mating Ring (P/N RS009667X, S/N 002, Damaged by Bearing Failure Posttest 092)

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Build 6 Disassembly. The spiral groove LOX seal was in good condition. Slight rub patterns were found on the mating ring and carbon face as shown in Fig. 81 and 82. A profile trace of carbon ring (Fig. 83) indicated slight wear near the inside diameter with a maximum depth of 0.00001 m (0.0004 in.). The spiral groove pad depth was 0.00001 m (0.0004 in.) pretst and posttest. The posttest static leakage was comparable with pretetest as shown in Table 9.

The helium seal was in good condition, except for carbon wear. The carbon ring recess pads were nearly worn away, as shown in Fig. 84 through 86. The profile traces indicated that the carbon ring wear ranged from 0.00001 to (0.0004 to 0.00002 m (0.0004 to 0.0008 in.) The carbon ring inside diameter was worn 0.000033 m (0.0013 in.). The static leakage at 206,843 N/m<sup>2</sup>g (30 psig) increased from an average of 0.024 m<sup>3</sup>/minute (1450 scim) pretest to 0.069 m<sup>3</sup>/minute (4200 scim) posttest (Table 11). The helium seal mating ring had a slight rub pattern on one side as shown in Fig. 87. The other side appeared clean, indicating possible rotation off center. Fig. 88 shows the profile trace across the rub pattern area, indicating grooving due to the carbon rings with a maximum depth of 0.0000038 m (0.000150 in.).

<u>Results and Conclusions</u>. A total of 34 tests for 1 hour, 40.2 minutes, on three tester builds was performed. Test leakage is shown in Fig. 89 indicating 0.119 to 0.139 m<sup>3</sup>/minute (4.2 to 4.9 scfm) at 2,068,427 N/m<sup>2</sup> (300 psi) and 8062 rad/s (77,000 rpm). Static  $GN_2$  leakage was higher than running, ranging from 0.221 to 0.374 m<sup>3</sup>/minute (7.8 to 13.2 scfm). No predicted  $GN_2$  leakages were available.

LIQUID OXYGEN TESTS

## Spiral Groove Piston Ring LOX Seal

<u>Build 7 Assembly</u>. The tester was reassembled with the same spiral groove LOX seal assembly. The carbon ring was lapped flat and the mating ring was lapped clean. A new helium seal assembly and mate ring was installed for the LOX testing.

<u>Tests 112-186</u>. The test objective was four 3-minute tests with liquid oxygen as the sealed fluid at speeds from 5235 to 8062 rad/s (50,000 to 77,000 rpm) and sealed pressures from 861,845 to 2,757,903 N/m<sup>2</sup>g to 400 psi), on the spiral groove LOX seal. Helium seal purge pressure was set at 206,843 N/m<sup>2</sup>g (30 psig) before and during each test.

A total of 38 tests for 42.75 minutes was required to achieve the first test point of 5236 rad/s (50,000 rpm) and 861,845 N/m<sup>2</sup>g (125 psig) sealed pressure for 3 minutes (Table 13). Facility problems included inadequate ranges of LOX seal upstream pressure control and drive turbine pressure control. LOX seal performance was good at 0.051 m<sup>3</sup>/minute (1.8 scfm) leakage and 3447 N/m<sup>2</sup>g (0.5 psig) drain pressure. The helium seal purge pressure was 209,601 N/m<sup>2</sup>g (30.4 psig) and total leakage was 0.181 m<sup>3</sup>/minute (6.4 scfm).

A total of 10 tests for 16.25 minutes was required to achieve the second test point of 8062 rad/s (77,000 rpm) and 1,034,214  $N/m^2g$  (150 psig) sealed pressure

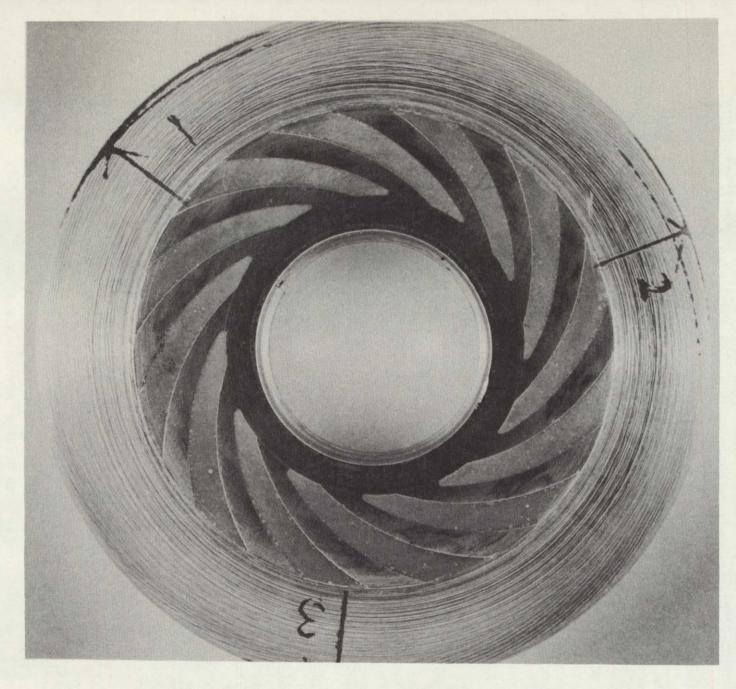


Figure 81. Spiral Groove LOX Seal Mating Ring (P/N CF-SP-41118, S/N 02, Posttest 111) ORIGINAL PAGE IS OF POOR QUALITY

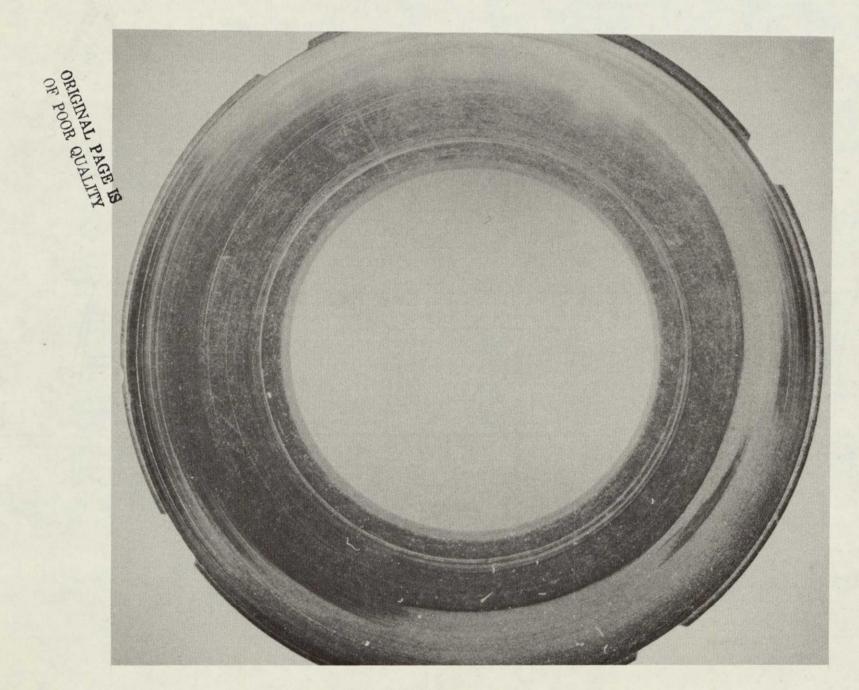


Figure 82. Spiral Groove LOX Seal Carbon Ring (P/N A28-0812-11, S/N 02, Posttest 111)

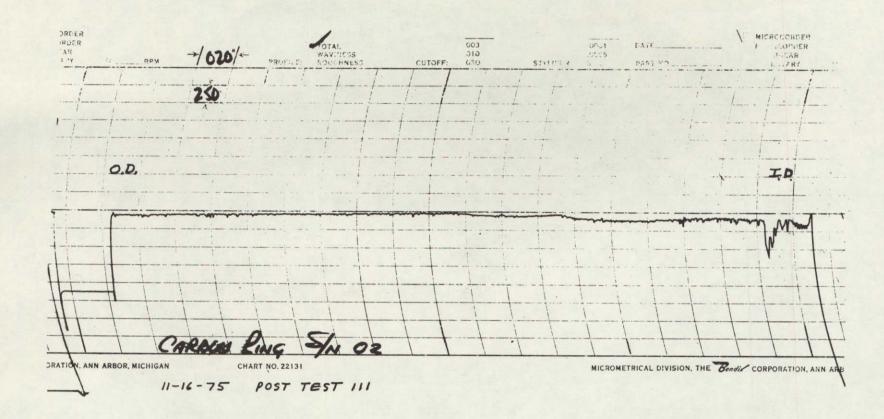


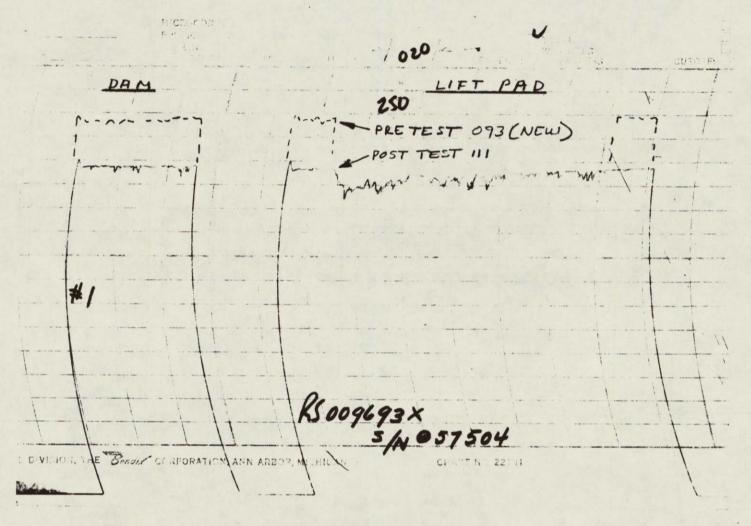
Figure 83. Typical Profile Trace of Spiral Groove LOX Seal Carbon Face (P/N A28-0812-11, S/N 02, Posttest 111)

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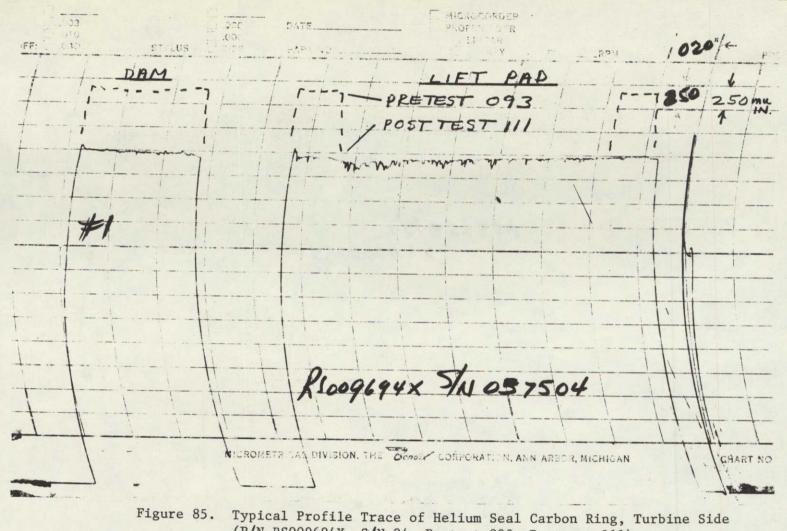
Figure 84. Typical Profile Trace of Helium Seal Carbon Ring, LOX Side (P/N RS009693X, S/N 04, Pretest 093 and Posttest 111)

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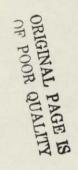
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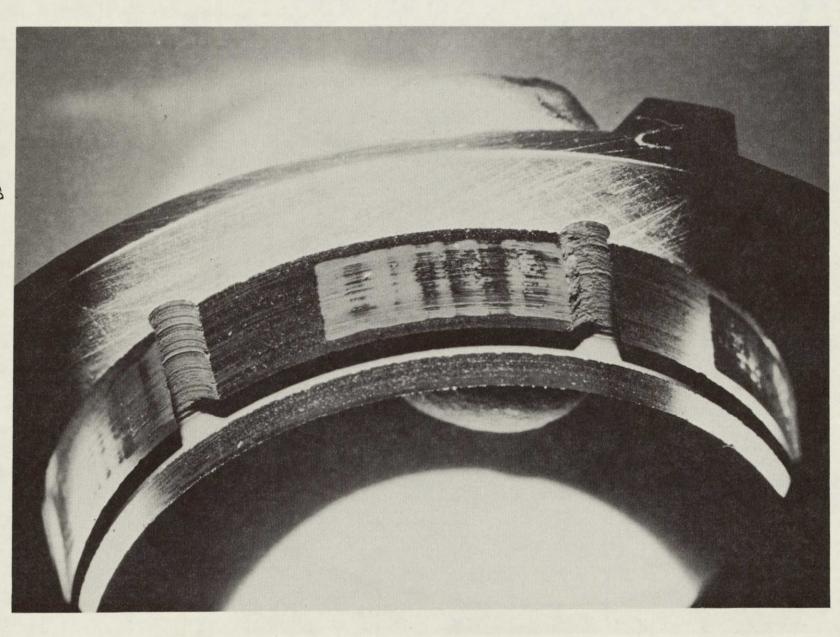
(P/N RS009694X, S/N 04, Pretest 093, Posttest 111)

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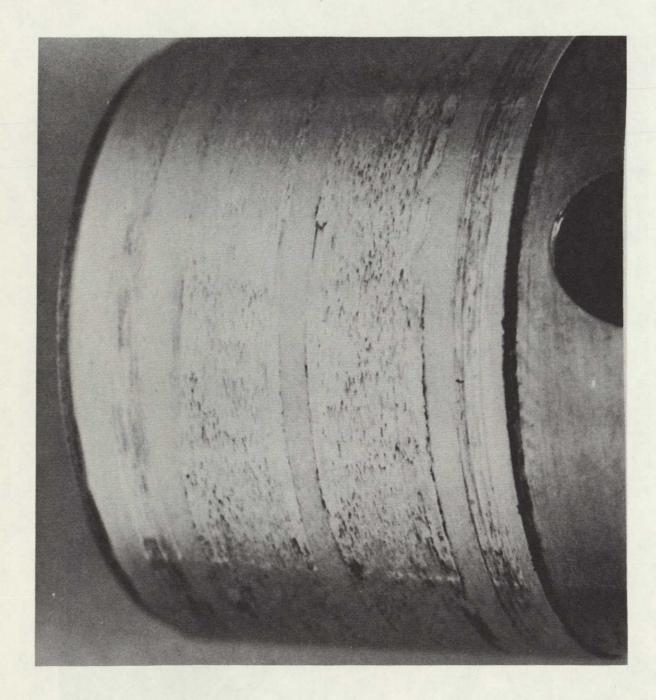
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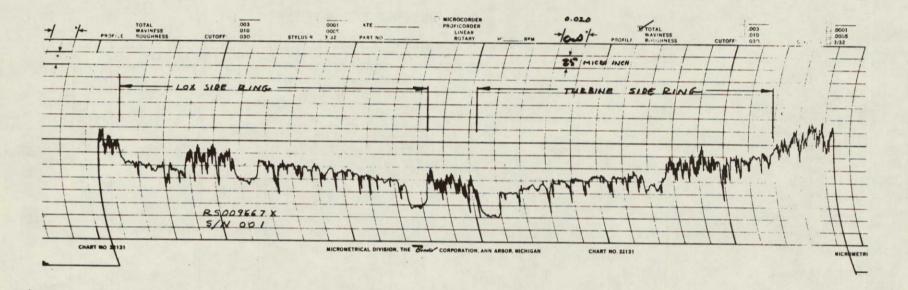
Figure 86. Helium Seal Turbine Side Ring (P/N RS009694X, S/N 04, Posttest 111)



TURBINE SIDE

LOX SIDE

Figure 87. Helium Mating Ring (P/N RS009667X, S/N 001, Posttest 111)



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Figure 88. Profile Trace of Helium Seal Mating Ring Surface at High Magnification (P/N RS009667X, S/N 02, Posttest 111)

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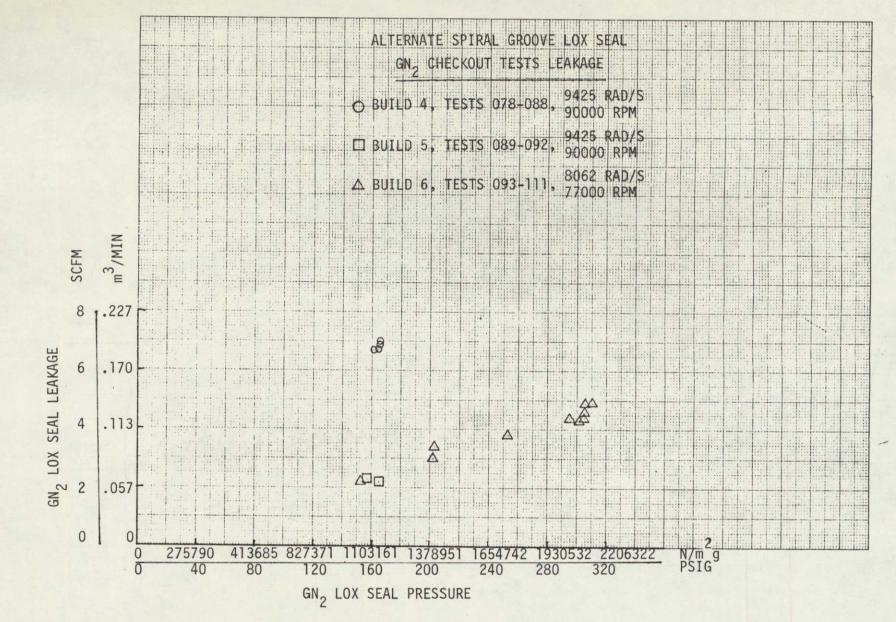


Figure 89. Test Leakage

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for 3 minutes (Table 13). The LOX seal leakage was less than the first test point at 0.040 m<sup>3</sup>/minute (1.4 scfm) with a drain pressure of 1379 N/m<sup>2</sup>g (0.2 psig). The helium seal purge pressure was 208,911 N/m<sup>2</sup>g (30.3 psig) and total leakage was 9.181 m<sup>3</sup>/minute (6.4 scfm).

A total of 76 tests for 1 hour, 24 minutes, was made on build 7. Nine tests were over 2 minutes duration and two of the test points were achieved. The tester control was very difficult. LOX seal leakage is shown in Fig. 90.

The tester was removed for inspection after high shaft torque and high seal leakages were experienced following an accidental dry spinup of the tester for approximately 20 seconds during an electrical checkout of the fast-start turbine drive system at Wyle Laboratories. The tester was extensively damaged by the dry spin.

Build 7 Disassembly. Tester shaft turning torque was high and rough, and static LOX seal leakage was excessive.

Tester disassembly revealed:

- 1. Helium seal and mating ring completely failed as shown in Fig. 91 due to high-speed rubbing and excessive shaft deflection.
- Spiral groove LOX seal carbon ring shattered, as shown in Fig. 92, due to excessive shaft deflection.
- Spiral groove LOX seal mate ring sheared the shaft key, as shown in Fig. 93; rotated on the pilot diameter, failing the static seal; and rubbed against the seal housing.
- 4. Both bearings were completely failed.

<u>Results and Conclusions</u>. Analysis indicates that the speed ran away when the turbine gas supply valve was opened at full turbine pressure of 1,034,215  $N/m^2g$  (150 psig) with the tester dry and unloaded.

The test data prior to the dry spin indicates normal tester operation. The LOX and helium seal leakages (Table 13) were normal on the last test (186) prior to the dry spin, indicating that the seals were in good condition at completion of testing. The seal leakage increased excessively after the dry spin without any additional tester rotation, except for hand torquing. Therefore, it was concluded that the seal damage occurred during the dry spin.

The test plan was revised to delete test points 3 and 4 of test schedule II. The test schedule II LOX testing was continued with 300 three-minute acceleration-deceleration tests from 0 to 8062 rad/s (77,000 rpm) and from 37,895 to 2,757,903  $N/m^2d$  (20 to 400 psid) LOX seal pressure using a new spiral groove LOX seal for a total time of 15 hours.

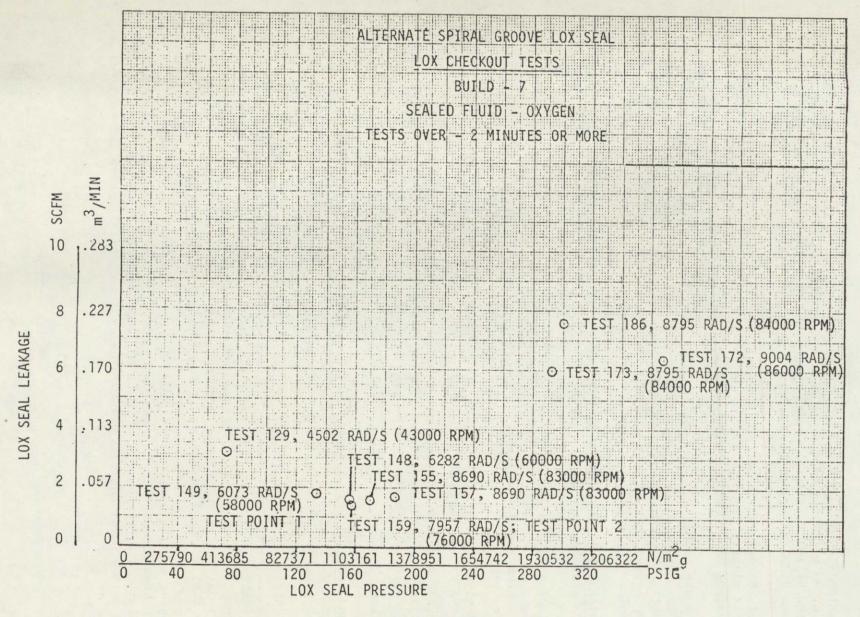


Figure 90. Seal Leakage

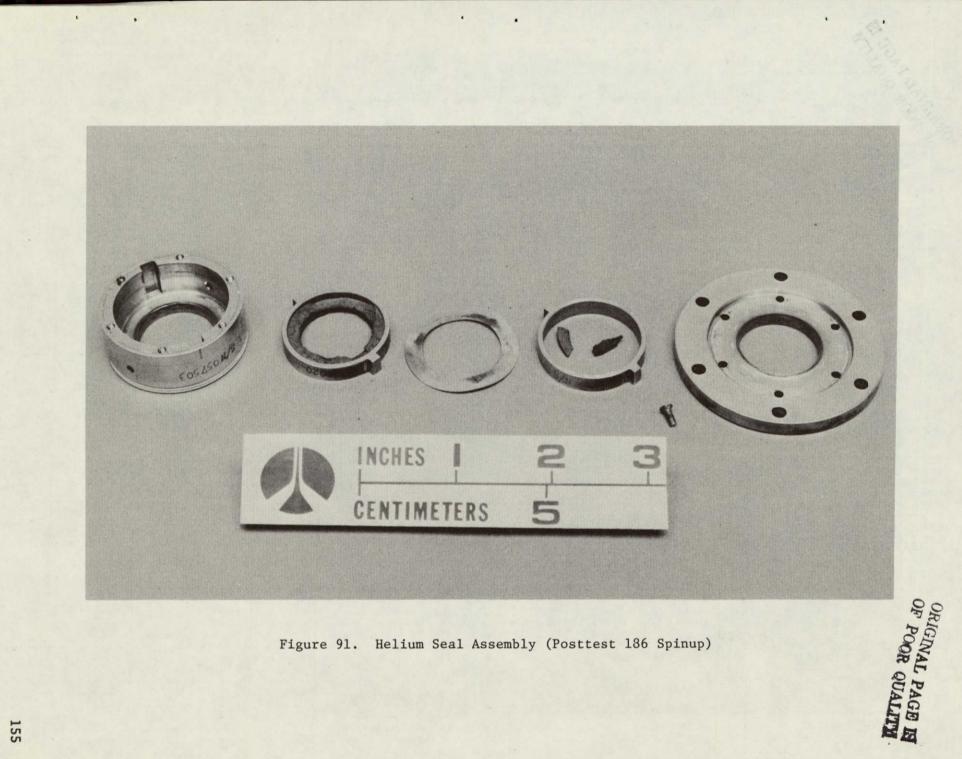


Figure 91. Helium Seal Assembly (Posttest 186 Spinup)

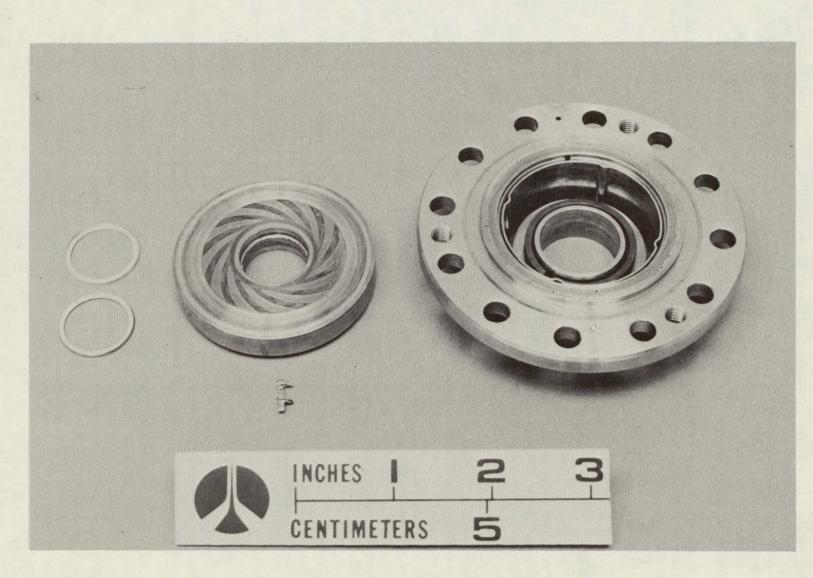


Figure 92. Spiral Groove LOX Seal Carbon Ring (Posttest 186 Spinup)

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Figure 93. Spiral Groove LOX Seal Mate, Key, Static Seal, and Housing (Posttest 186 Spinup)

<u>Build 8 Assembly</u>. The tester assembly was completed with a new spiral groove alternate design LOX seal. The installed length spring load was 16.24 N (3.65 pounds). The spiral groove mating ring was nicked in the seal dam area during removal when balancing the rotating hardware. The mating ring was relapped prior to assembly; however, the nick was not removed. The LOX seal  $GN_2$  static leakage was 0.374 m<sup>3</sup>/minute (13.2 scfm) at 2,068,427 N/m<sup>2</sup>g (300 psig).

A new helium purged intermediate seal was installed with 0.000028 m (0.0011 in.) ambient diametral clearance on the LOX side and 0.000033 m (0.0013 in.) on the turbine side. The static helium leakage at 206,843 N/m<sup>2</sup>g (30 psig) was 0.015 m<sup>3</sup>/minute (0.52 scfm) for the LOX side and 0.031 m<sup>3</sup>/minute (1.09 scfm) on the turbine side. The LOX and helium seal build data are listed in Tables 9 and 11.

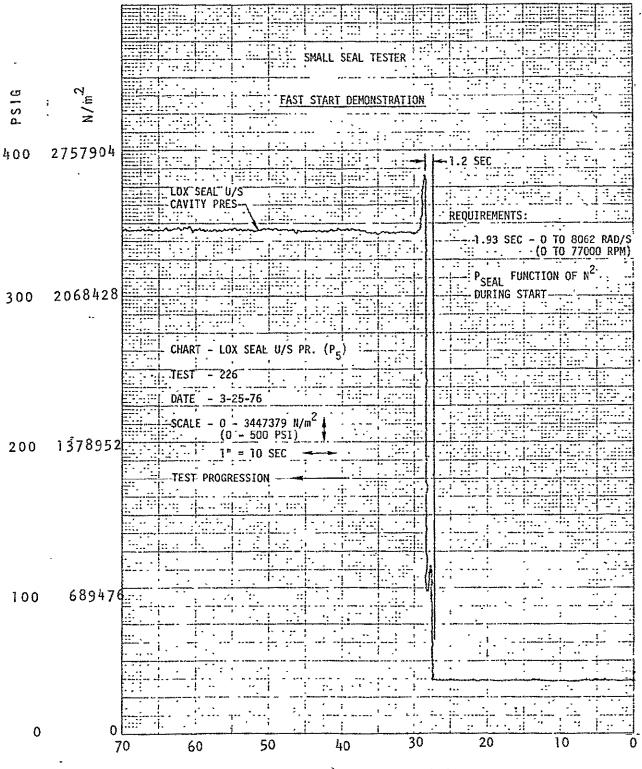
Tests 187-332. The test objective was 150 starts, or a total time of 5 hours on the spiral groove LOX seal. The speed was ramped from 0 to 8062 rad/s (77,000 rpm) in 2 seconds using the turbine fast-start system. The seal pressure was ramped from 206,843 to 2,757,903 (30 to 400 psig) in the same time. The tester was stopped and the seal pressure was vented in less than 4 seconds using the turbine reversing nozzles.

The initial tests (187-227) were facility checkout tests. Limited turbine pressure capability prevented reaching test point conditions for tests 187-202. The final two tests (226 and 227) were satisfactory duration tests, completing the facility checkout. The fast-start speed and LOX seal pressure start transients are shown in Fig. 94 and 95. The LOX seal pressure ramp rate as a function os speed is shown in Fig. 96. The shutdown transient speed and LOX seal pressure data are shown in Fig. 97 and 98. The deceleration requirements were met without using the tester brake system.

Tests 228 through 327 included 79 duration tests and premature cuts due to problems with operation of the turbine valves. Tests 328 through 331 were terminated due to problems with the turbine drive valves.

Test 332 was cut during the fast start by a bearing cavity overpressure and tester accelerometer redline. The bearing drain temperature was offscale warm 144.3 K (-200 F). Posttest inspection revealed carbon debris in the turbine side drain ports. The tester shaft torque was in excess of 1.695 N·m (15 in.-1b) compared to previous values of less than 0.11299 N·m (1 in.-1b). The inspection and test data indicated a bearing failure with damaged seals.

A summary of the steady-state LOX and helium seal leakages is presented in Table 13. The LOX seal leakage varied from 0.544 to 1.441 m<sup>3</sup>/minute (19.2 to 50.9 scfm). The LOX seal leakage on the short tests was generally higher (up to approximately 1.982 m<sup>3</sup>/minute; 70 scfm) during the start transient. The steady-state leakage stabilized at approximately 0.850 to 1.416 m<sup>3</sup>/minute (30 to 50 scfm) after the first half hour of testing. The helium seal leakage held steady during the test series at approximately 0.396 to 0.453 m<sup>3</sup>/minute (14 to 16 scfm).



## TIME, SECONDS

Figure 95. Small Seal Tester

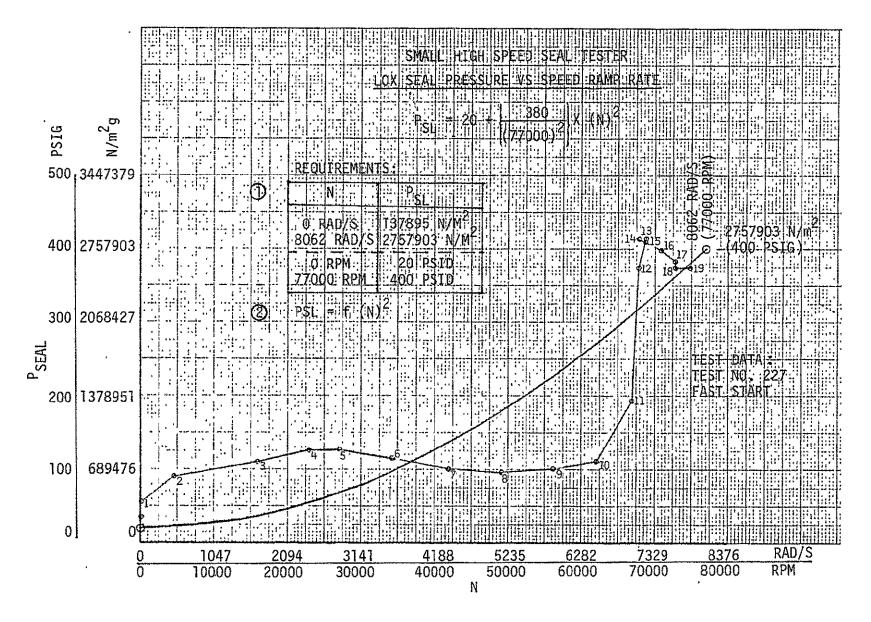


Figure 96. Seal Pressure vs Speed for Acceleration Testing

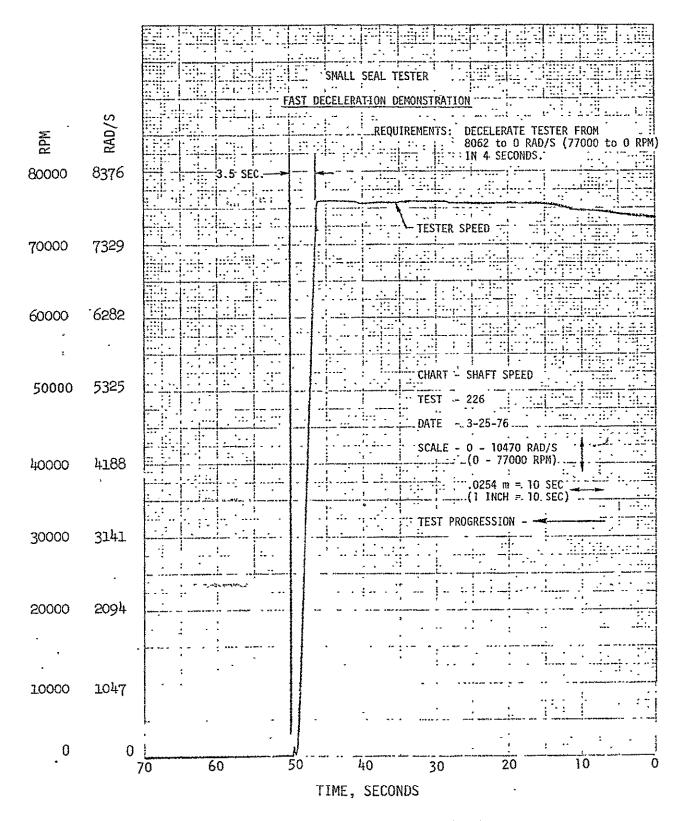
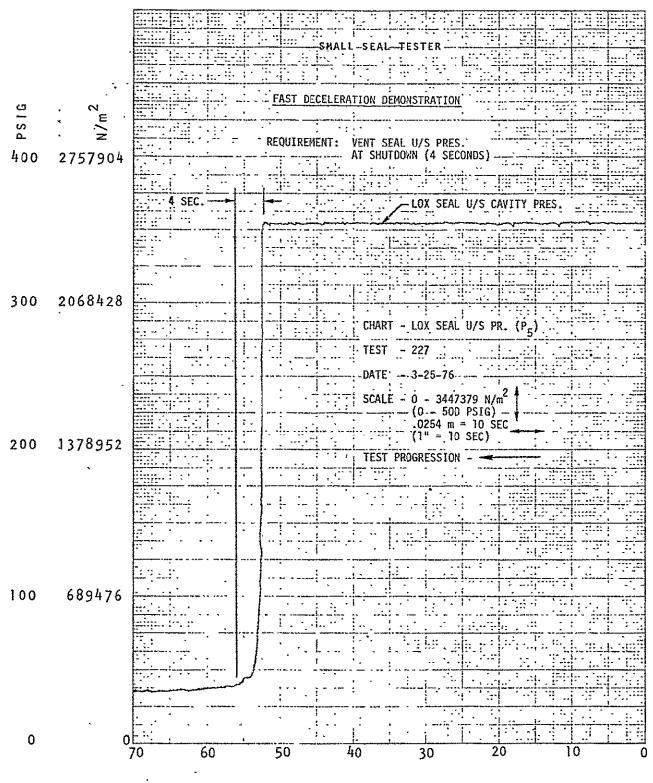


Figure 97. Deceleration Testing (rpm)



## TIME, SECONDS

Figure 98. Deceleration Testing (pressure)

<u>Build 8 Disassembly</u>. Inspection revealed both bearings to be completely failed. The spiral groove LOX seal carbon seal ring was shattered from the shaft thrust load after the bearings failed. The spiral groove mating ring surface was heavily rubbed and scored by the excessive thrust load. Since the seal leakage was consistent on the tests prior to the bearing failure, it is concluded that the mating ring rubbing occurred after the bearing failed.

The helium seal carbon rings and mating ring sleeve were heavily rubbed and worn from excessive misalignment after the bearings failed. The seal housing was damaged by rubbing contact at the mating ring.

The damaged seal hardware is shown in Fig. 99 through 104.

## Rayleigh Step Piston Ring LOX Seal

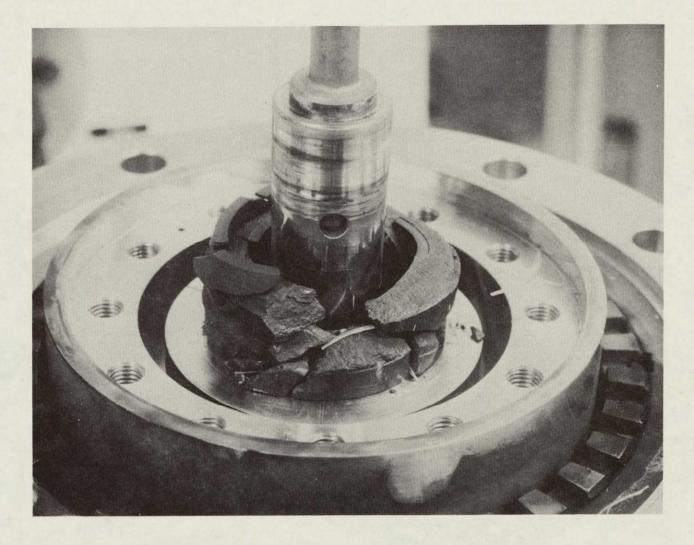
<u>Build 9 Assembly</u>. A new Rayleigh step LOX seal was installed with a new mating ring. The seal was assembled with four springs for an installed load of 9.207 N (2.07 pounds). The spring load was reduced from the 12.454 N (2.8 pounds) load used on the last Rayleigh step seal tested to reduce wear. The seal static leakage was 0.018 m<sup>3</sup>/minute (1090 scim) at 2,068,427 N/m<sup>2</sup>g (300 psig).

A used helium seal from build 6 was installed. The carbon lift pads were partially worn; however, the seal was in satisfactory condition. The recess pad depth varied from 0 to 0.000013 m (0.0005 in.). The seal ring diametral clearance was 0.00005 m (0.0021 in.) on the LOX side and 0.000061 m (0.0024 in.) on the turbine side. The static He leakage at 206,843 N/m<sup>2</sup>g (30 psig) was 0.065  $m^3/minute$  (3980 scim) on the LOX side and 0.080  $m^3/minute$  (4900 scim) on the turbine side.

<u>Tests 333-412</u>. The test objective was to complete a minimum of 75 starts for a total time of 2.5 hours on the Rayleigh step LOX seal prior to a scheduled in-spection. The scheduled test duration was 3 minutes. The tests were part of the 300 starts and 10 hours required for the schedule II fast-start LOX testing. The speed was ramped from 0 to 8062 rad/s (77,000 rpm) in 2 seconds.

The LOX seal pressure was ramped from 206,843 to 2,757,903  $N/m^2g$  (30 to 400 psig) in the same time. A total of 80 tests for 2.5 hours was performed to complete the test objective (Table 13). The initial tests (333 through 339) were all over-speed cutoffs with indicated maximum speeds in excess of 10,470 rad/s (100,000 rpm). The overspeed cutoffs were apparently caused by the tester drag torque being lower, since the turbine start pressures were the same as the last build. The turbine start pressure was lowered and most of the remaining tests were scheduled duration. A total of 50 duration tests (3 minutes) were performed.

The steady-state LOX seal leakage was consistent from the beginning of the test series until the end at approximately 0.793 to 0.850  $m^3/minute$  (28 to 30 scfm), indicating no deterioration of the sealing surfaces. The LOX seal drain pressure also was consistent at approximately 27,579 to 34,474 N/m<sup>2</sup>g (4 to 5 psig).



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Figure 99. Shattered Spiral Groove LOX Seal Following Bearing Failure (Build 8, Posttest 332)

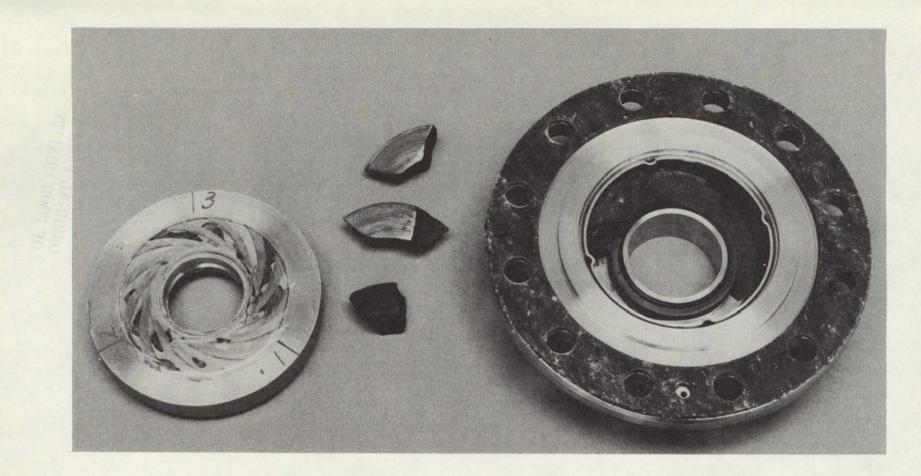
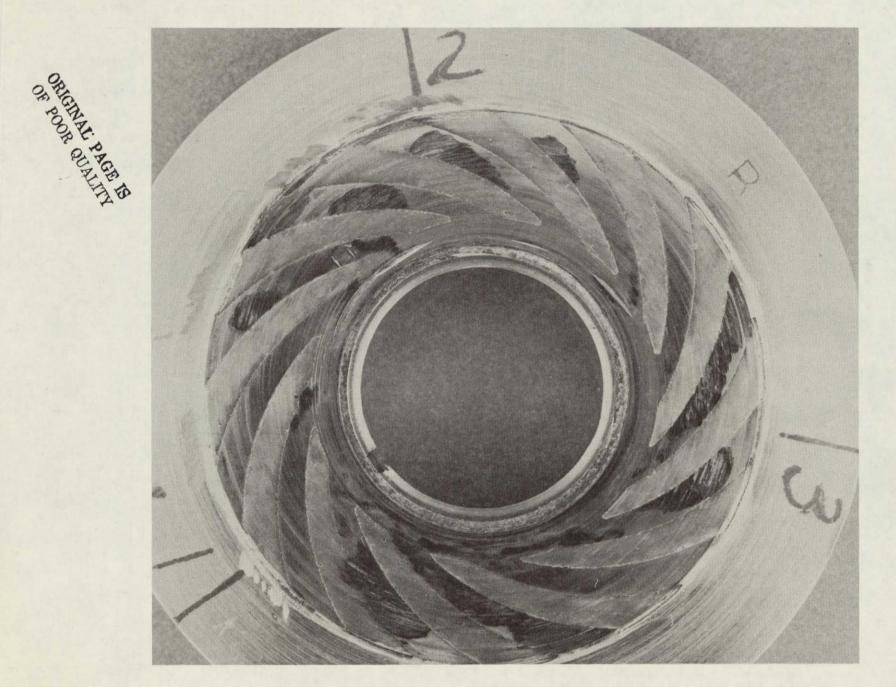


Figure 100. Spiral Groove LOX Seal Mate Ring, Some Carbon Pieces, and Housing (Build 8, Posttest 332)

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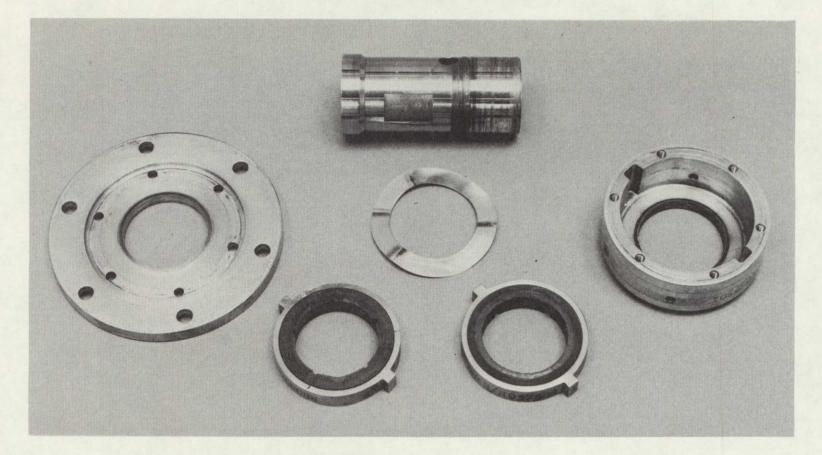
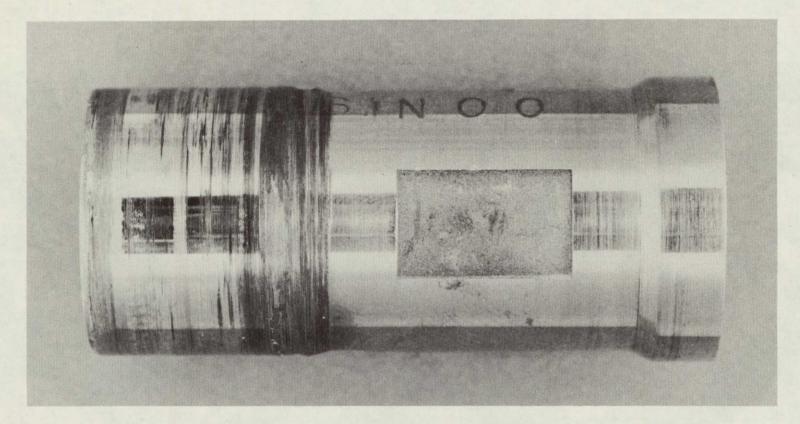


Figure 102. Helium Seal Assembly Following Bearing Failure (Build 8, Posttest 332)

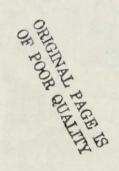
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Figure 103. Helium Seal Mate Ring (Build 8, Posttest 332)



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TURBINE SIDE

LOX SIDE

Figure 104. Helium Seal Carbon Rings (Build 8, Posttest 332)

<u>Build 10 Disassembly</u>. The scheduled inspection after an additional time of 2.61 hours for a total of 5.12 hours revealed the Rayleigh step piston ring LOX seal to be in excellent condition. The carbon face was polished with very slight circumferential scratches from minute particles in the sealed fluid. There was no measurable wear or deterioration. The mating ring surface had a uniform polished trace in the area of the seal dam and hift pads, indicating that the mating ring and seal ring surfaces are remaining flat during operation. The posttest static leakage was approximately the same as pretest (Table 9 ).

The helium seal condition was the same as pretest 413. There was no significant additional wear on the carbon seal rings. The mating ring surface appeared the same as pretest with a heavy contact pattern, but no measurable wear. The measured posttest static leakage was less than pretest (Table 11).

<u>Build 11 Assembly</u>. The tester was reassembled with the same seal hardware without any rework. The Rayleigh step LOX seal static  $GN_2$  leakage was 0.027 m<sup>3</sup>/minute (1625 scim) at 2,068,427 N/m<sup>2</sup>g (300 psig) (Table 9).

The helium seal ring diametral clearance was 0.000/m (0.0045 9n.) on the LOX side and 0.00013 m (0.0050 in.) on the turbine side. The static healium leak-age at 206,843 N/m<sup>2</sup> (30 psig) was 0.123 m<sup>3</sup>/minute (7500 scim) on the LOX side and 0.146 m<sup>3</sup>/minute (8900 scim) on the turbine side (Table 11).

Tests 518-559. The test objective was to complete a minimum of 75 fast-start tests for a total time of 2.5 hours on the Rayleigh step LOX seal.

A total of 42 starts for 1.2 hours was performed (Table 13). The scheduled test duration was 3 minutest. Nineteen of the tests were of scheduled duration. The remainder were terminated due to speed control and other operation problems. The last test was terminated when the tester vibration level increased to 6 g(p-p) with spikes up to 10 g(p-p). The tester was removed for inspection due to the high vibration level.

The LOX seal leakage varied from approximately 0.566 to 0.850 m<sup>3</sup>/minute (20 to 30 scfm) at 2,757,903 N/m<sup>2</sup>g (400 psig). The LOX seal drain pressure varied from 19,995 to 39,990 N/m<sup>2</sup>g (2.9 to 5.8 psig). The leakage and drain pressure during this test series was approximately the same as the initial test series when the seal was new.

The helium seal leakage varied from 0.265 to 0.283  $m^3$ /minute (9.35 to 10 scfm) to approximately 206,843 N/m<sup>2</sup>g (30 psig) purge pressure. The leakate has not changed significantly during the last 6.34 hours.

<u>Build 11 Disassembly</u>. The tester inspection revealed the hardware to be in good condition. The bearing race wear tract indicated that the bearings had run unloaded during part of the testing. The bearings were in good condition otherwise. Measurements indicated that the bearing preload spring compression had decreased slightly. It was concluded that the high vibration was caused by the bearings unloading due to the lower preload and high bearing cavity pressure. The LOX seal condition did not appear to change during the additional 1.2 hours. The carbon face was polished with very slight circumferential scratches and no measureable wear. The mating ring surface also appeared the same with a uniform polished trace in the area of the seal dam and lift pads. The posttest static leakage was approximately the same as pretest (Table 9 ).

The helium seal and mating ring condition has not changed since the initial wear which occurred on builds 6 and 9 (Fig. 86 and 87 ). The posttest static leakage was nearly the same as pretest (Table 11).

Build 12 Assembly. The tester was reassembled with the same seal hardware without any rework. The static leakages are given in Tables 9 and 11.

Tests 560-632. The test objective was to complete 75 fast start tests for a total time of 2.5 hours on the Rayleigh step LOX seal.

A total of 72 starts for 2.58 hours was performed (Table 13) to complete the test objective. Twelve of the tests were of scheduled duration. Steady-state conditions for 1 minute or longer were attained on 42 of the tests. The scheduled test duration was increased from 3 minutest to 6 minutes and then to 10 minutes to accumulate the required starts and total time. Numerous tests were terminated prematurely due to tester overspeed and other problems.

The LOX seal leakage was consistent at approximately  $0.850 \text{ m}^3/\text{minute}$  (30 scfm) during the entire test series. The LOX seal drain pressure varied from approximately 34,474 to 41,369 N/m<sup>2</sup>g (5 to 6 psig). The leakages and drain pressures are continuing to be consistent with the initial test series when the seal was new.

The helium seal leakage increased on this build to approximately 0.425 m<sup>3</sup>/ minute (15 scfm) at 206,843 N/m<sup>2</sup>g (30 psig) purge pressure compared to approximately 0.283 m<sup>3</sup>/minute (10 scfm) on the prior build (No. 11).

<u>Build 12 Disassembly</u>. The scheduled tester inspection revealed the hardware to be in good condition, except for additional wear on the helium seal. The LOX side carbon ring was worn an additional 0.0002 m (0.008 in. diametral. The turbine side carbon ring was worn an additional 0.00028 m (0.011. in.) diametral. The helium mating ring was grooved approximately 0.000005 m (0.0002 in.) on the LOX side and 0.0000025 m (0.0001 in.) on the turbine side. The mating ring wear was on one side only, indicating eccentric rotation.

The helium seal static leakage decreased from 0.271  $m^3$ /minute (9.55 scfm) pretest to 0.197  $m^3$ /minute (6.94 scfm) posttest, indicating that the additional wear did not have a significant effect on the seal static leakage.

The LOX seal was in excellent condition with no measurable wear. The pretest and posttest static leakages were comparable (Table 9 ).

Build 13 Assembly. The tester was reassembled with the same seal hardware without any rework. The static leakages are given in Tables 9 and 11.

Tests 633-665. The test objective was to complete the required 300 starts and 10 hours on the Rayleigh step LOX seal. An additional 1.1 hours was required.

A total of 33 starts for 1.2 hours was performed to complete the test objective (Table 13). The scheduled duration of 3 minutes was achieved on 22 tests. The remainder were terminated prematurely due to tester overspeed and other problems.

The LOX seal leakage was steady at 0.736 to 0.821  $m^3$ /minute (26 to 29 scfm). The LOX seal drain cavity pressure varied from 31,026 to 39,300 N/m<sup>2</sup>g (4.5 to 5.7 psig). The leakage rates and drain pressures after 10 hours of testing are the same as the first test when the seal was new. The seal performance has been very consistent and repeatable through the entire test series. There has been no measurable degradation in performance.

The helium seal leakage increased on this build (No. 13) to approximately 0.595  $m^3/mixture$  (21 scfm) at 206,843 N/m<sup>2</sup>g (30 psig) purge pressure compared to approximately 0.425 m<sup>3</sup>/minute (15 scfm) on the last build (No. 12) and approximately 0.142 m<sup>3</sup>/minute (5 scfm) when new. The increase in leakage rate corresponds to the measured carbon ring wear.

<u>Build 13 Disassembly</u>. The scheduled tester inspection after an additional time of 1.2 hours for a total accumulated time of 10.12 hours on the Rayleigh step LOX seal revealed the LOX seal to be in excellent condition with no measurable wear or deterioration. The carbon face was polished with very slight circumferential scratches. The surface profile traces (Fig. 105) indicate that the recess pad depth and sealing dam height are the same as new.

The LOX mating ring surface has a polished mark in the area of the sealing dam and faint traces of contact in the area of the lift pads. The mating ring surface profile trace (Fig. 106) shows the polished area at the dam with\_no significant wear.

The LOX seal secondary element segmented carbon piston ring was in good condition with no visible wear or deterioration.

The helium seal carbon rings and the mating ring surface are worn. The measured wear did not change since the build 12 inspection. The carbon rings have worn a total of 0.00030 m (0.0118 in.) diametral on the LOX side and 0.00034 m (0.0135 in.) diametral on the tyrbine side since new. The mating ring is grooved approximately 0.000005 m (0.0002 in.) on the LOX side and 0.0000025 m (0.0001 in.) on the turbine side (Fig. 107).

The seal hardware condition is shown in Fig. 108 through 116.

One LOX seal assembly was utilized for the LOX acceleration tests for 332 starts and 10 hours, 18 minutes of accumulated time. No LOX checkout tests were made. LOX seal leakage versus test time is shown in Fig. 117.

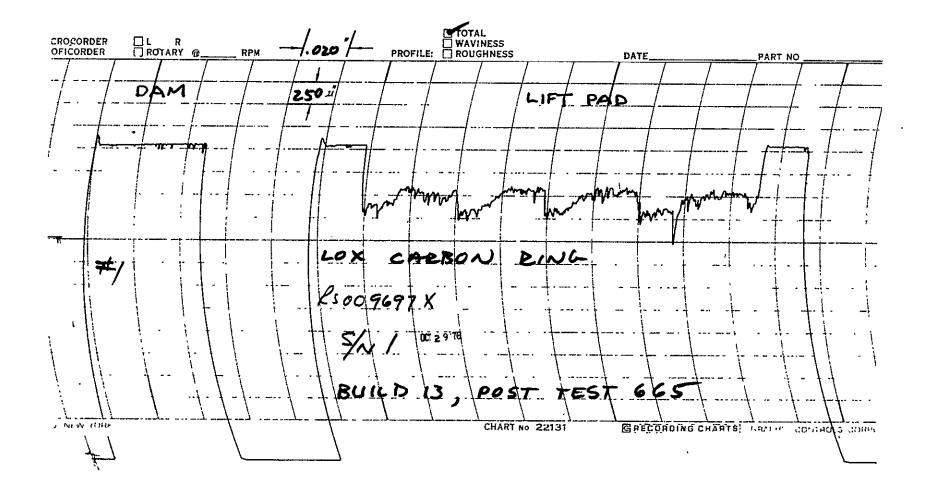


Figure 105. Typical Surface Profile Trace LOX Seal Carbon Ring (RS009697X, S/N 01, Posttest 665, Build 13, After 332 Starts and 10.12 Hours)

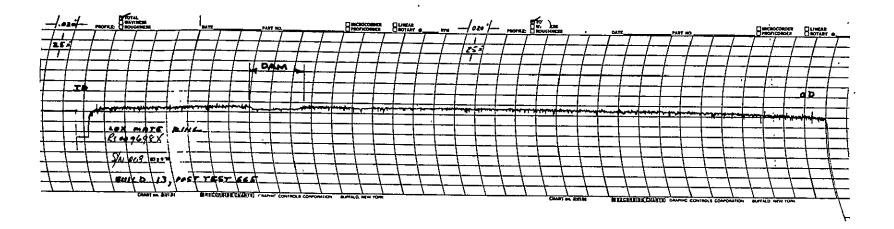
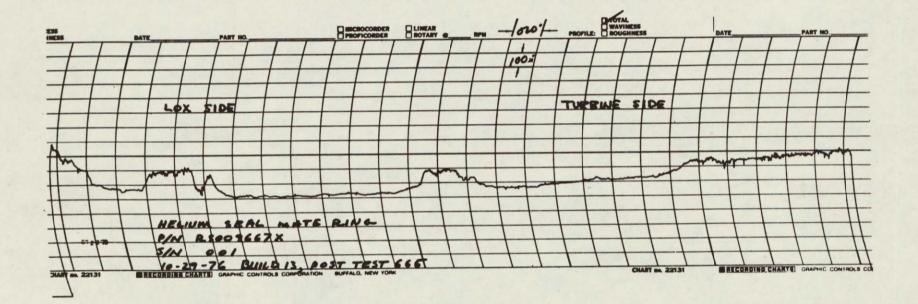


Figure 106. Surface Profile Trace LOX Seal Mating Ring (RS009698X, S/N 03, Build 13, Posttest 665, After 332 Starts and 10.12 Hours)

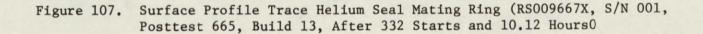


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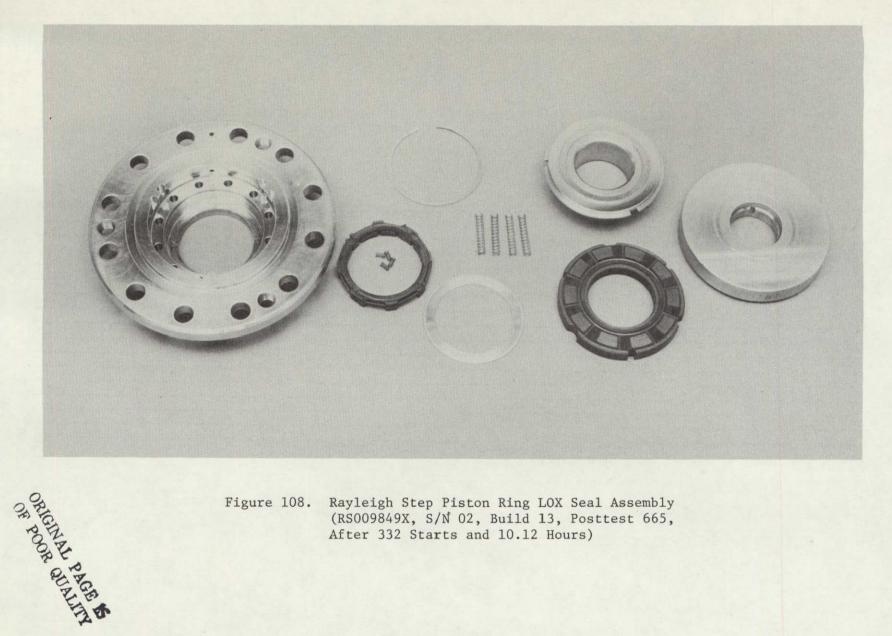
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Rayleigh Step Piston Ring LOX Seal Assembly (RS009849X, S/N 02, Build 13, Posttest 665, After 332 Starts and 10.12 Hours) Figure 108.

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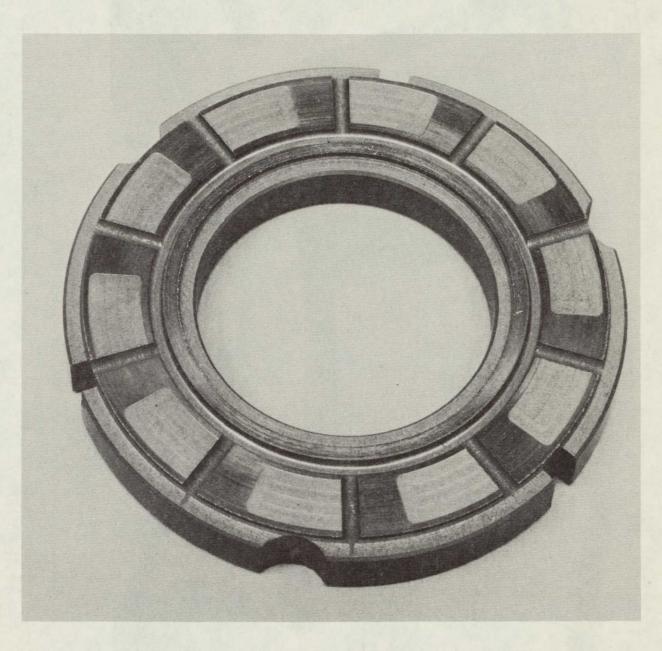


Figure 109. Rayleigh Step LOX Seal Carbon Ring (RS009697X, S/N 01, Build 13, Posttest 665, After 332 Starts and 10.12 Hours)

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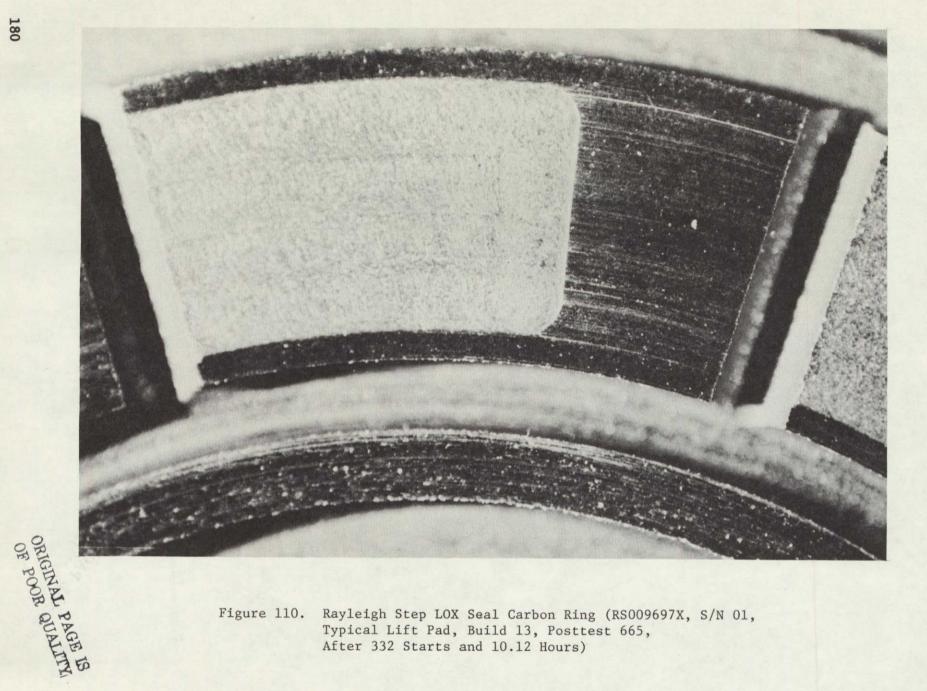


Figure 110. Rayleigh Step LOX Seal Carbon Ring (RS009697X, S/N 01, Typical Lift Pad, Build 13, Posttest 665, After 332 Starts and 10.12 Hours)

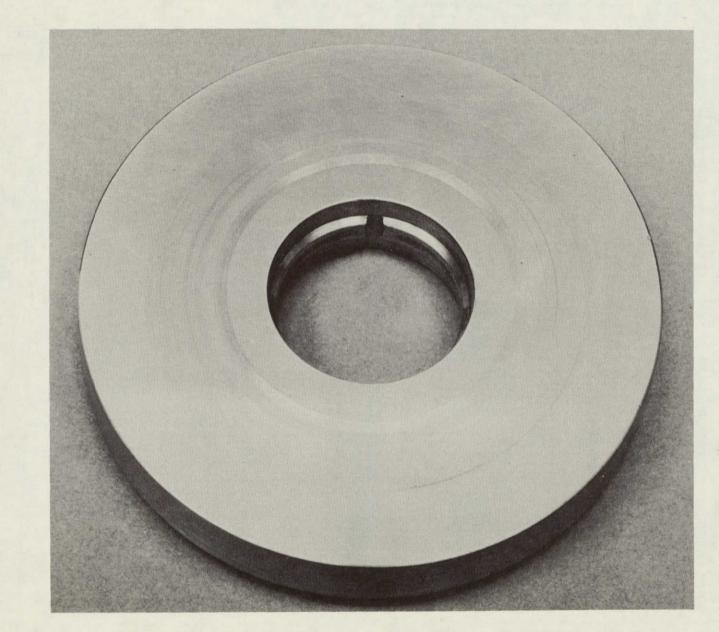


Figure 111. Rayleigh Step LOX Seal Mating Ring (RS009698X, S/N 03, Build 13, Posttest 665, After 332 Starts and 10.12 Hours)

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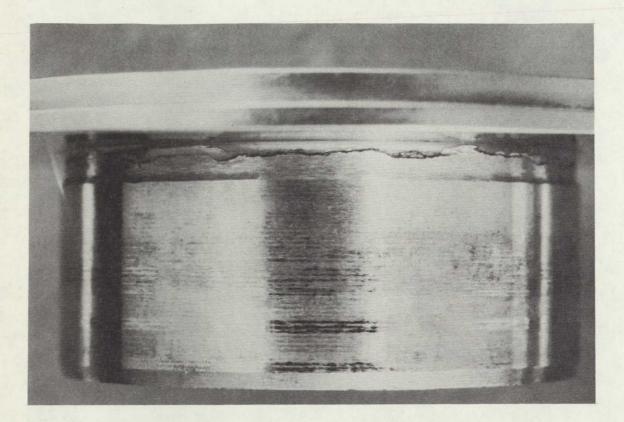


Figure 112. Piston Ring LOX Seal Pilot Ring (RS009848X, S/N 02, Build 13, Posttest 665, After 332 Starts and 10.12 Hours)

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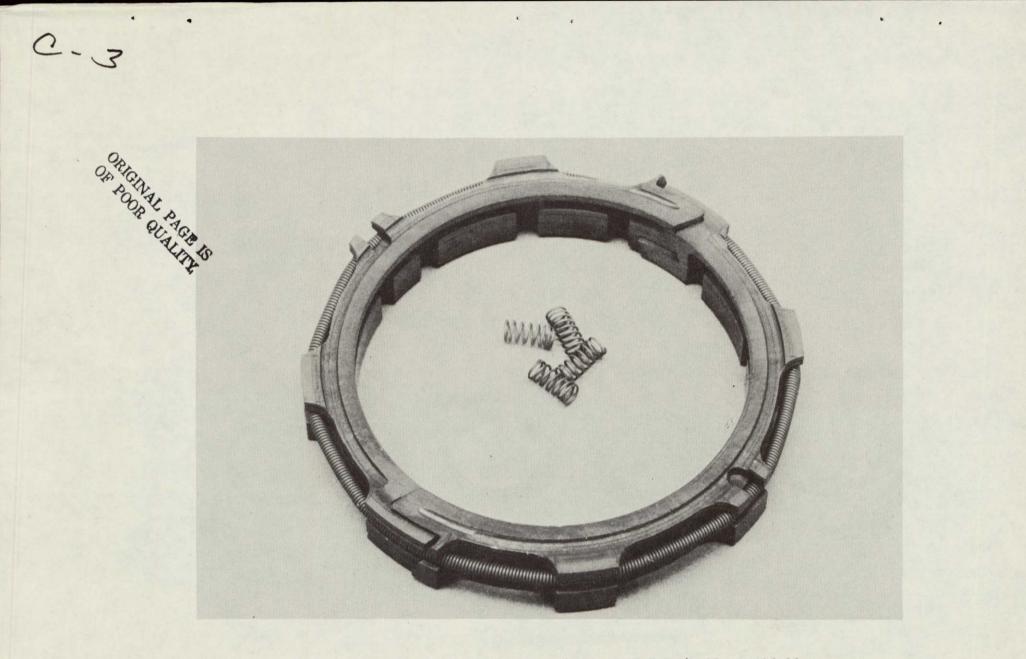


Figure 113. LOX Seal Piston Ring (SSCY 5097-8, S/N 02, Build 13, Posttest 665, After 332 Starts and 10.12 Hours)

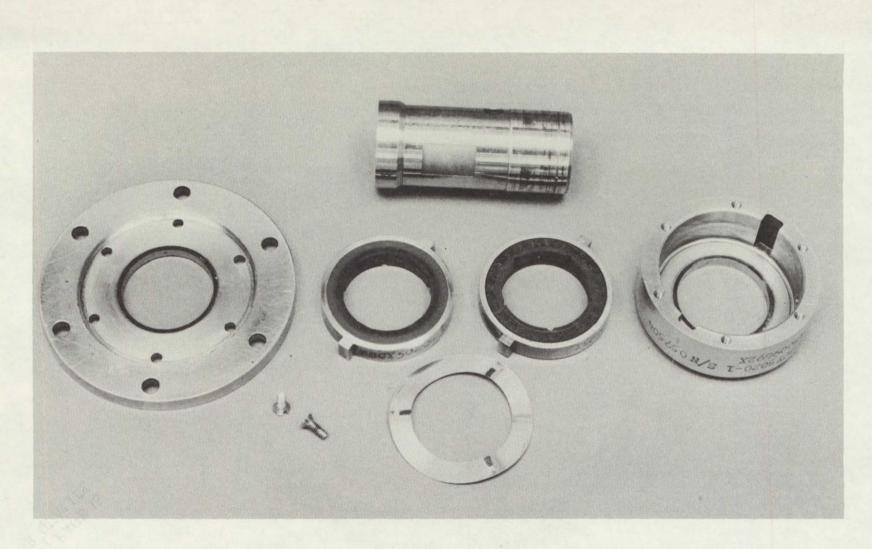
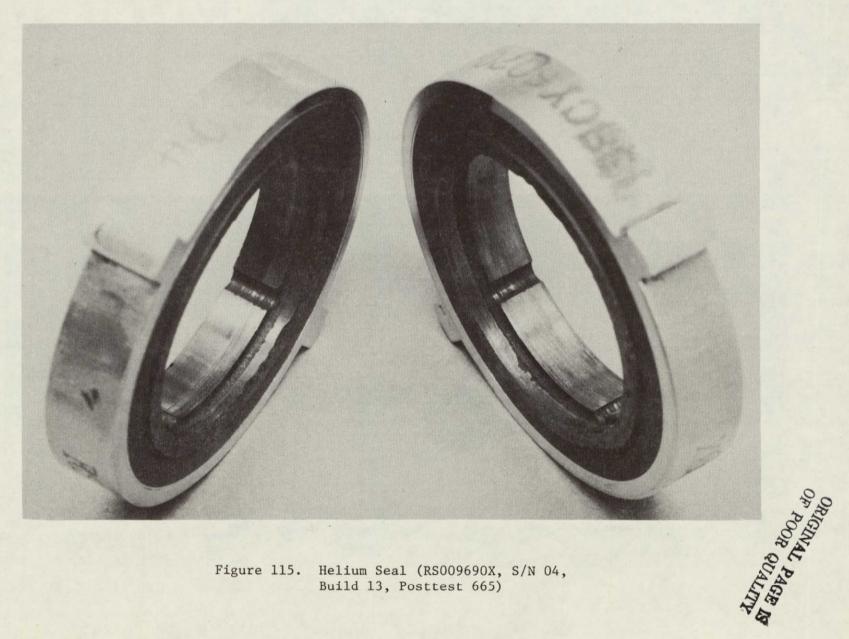


Figure 114. Helium Seal Assembly (RS009690X, S/N 04, Build 13, Posttest 665)



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Figure 115. Helium Seal (RS009690X, S/N 04, Build 13, Posttest 665)

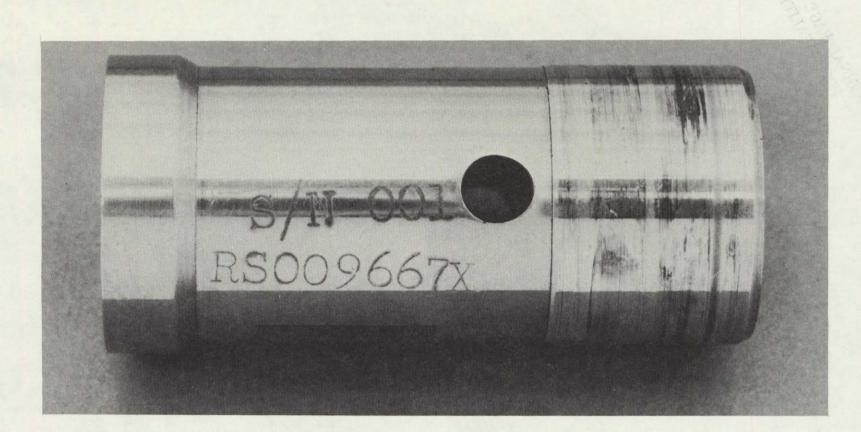


Figure 116. Helium Seal Mating Ring (RS009667X, S/N 001, Build 13, Posttest 665)

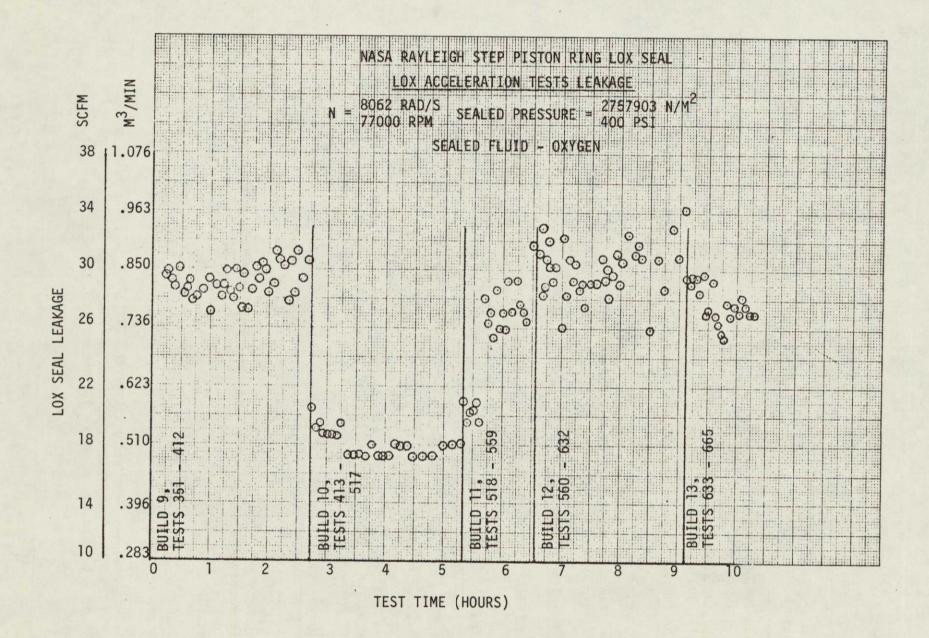


Figure 117. LOX Seal Leakage

Predicted LOX seal leakages are compared with test leakage below:

Speed, rad/s (rpm) and pressure, N/m <sup>2</sup> (psi)	Liquid Oxygen, Predicted, m <sup>3</sup> /minute (scfm)	Gaseous Oxygen Predicted, m <sup>3</sup> /minute (scfm)	Oxygen Test m <sup>3</sup> /minute (scfm)
9425 (90,000 3,206,062 (465)	1.873 (66.13)	0.164 (5.78)	
8062 (77,000 2,757,903 (400)			0487 to 0.934 (17.2 to 33.0)
6282 (60,000) 2,171,848 (315)	1.085 (38.30)	0.112 (3.97)	

 $GN_2$  static leak check leakage at 2,068,427 N/m<sup>2</sup> (300 psi) ranged from 0.017 to 0.027 m<sup>3</sup>/minute (0.59 to 0.94 scfm) for the five builds.

## Spiral Groove Piston Ring LOX Seal

<u>Build 14 Assembly</u>. The tester was assembled with a reworked spiral groove LOX seal and a new Rayleigh step floating ring helium seal. The spiral groove mating ring from build 8, which was damaged by rubbing contact when the tester bearings failed, was lapped slightly to partially clean up the surface scoring. Approximately 0.0000025 m (0.0001 in.) was removed from the surface to reduce the spiral groove depth from 0.00001 m (0.0004 in.) to 0.000008 m (0.0003 in.).

The LOX seal was assembled with a new carbon seal ring and a new carbon seal ring and a new revised piston ring. The new piston ring had the pressure balance groove and vent slot relocated to eliminate a direct leak path. A wave spring around the piston ring was utilized to seat the ring. The housing and spring from build 8 was used.

The new helium seal was manufactured by a different supplier (Clevite). The recess pad depths manufactured by a different supplier (Clevite). The recess pad depths were irregular (Fig. 118), with a variation from 0.0000025 to 0.0000033 m (0.0001 to 0.0013 in.) compared to the specification values of 0.000013 to 0.0002 m (0.0005 to 0.0008 in.). The mating ring from build 13 was ground down and replated with chrome. The seal ring diametral clearances at assembly were 0.000036 m (0.0014 in.) on the LOX side and 0.00004 (0.0016 in.) on the turbine side.

Tests 666-749. The test objective was 80 fast-start LOX tests and 4 hours on the spiral groove LOX seal.

A total of 84 starts for 4.16 hours was performed to complete the test objective (Table 13). The scheduled duration of 3 minutes was achieved on 64 tests. The duration was increased to 10 minutes on 6 tests to accumulate additional time.

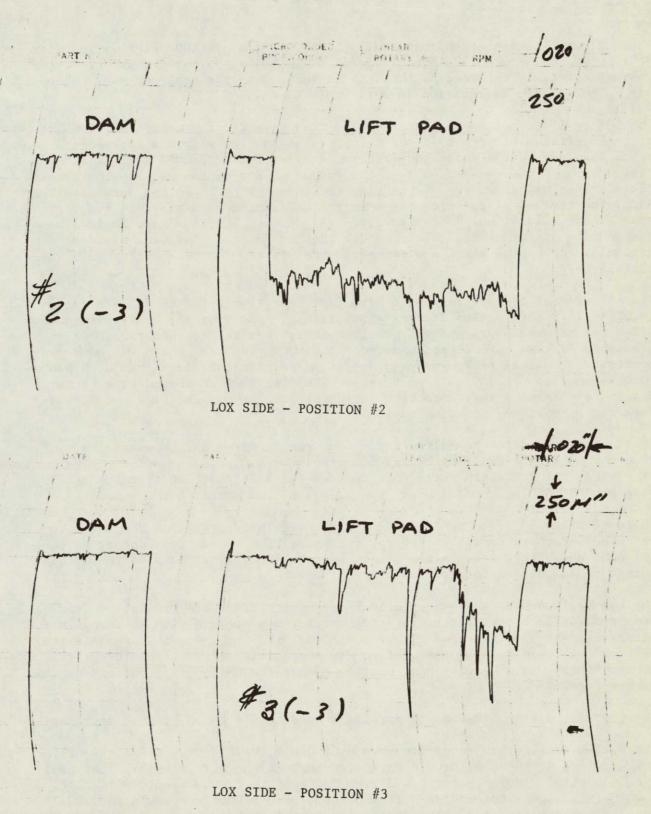


Figure 118. Surface Profile Traces of Helium Seal (RS009690X, S/N 05, Recess Lift Pads, Pretest 66, Build 14)

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The LOX seal leakage at approximately  $(2,757,903 \text{ N/m}^2\text{g})$  (400 psig) varied from 0.510 to 0.765 m<sup>3</sup>/minute (18 to 27 scfm). The leakage after 4 hours of testing was approximately the same as the initial tests when the seal was new. There was no measurable degradation in seal performance.

The LOX seal did not seal at static condition when first pressurized with LOX at the beginning of each test series. The carbon seal ring was apparently stuck open due to the increased friction drag of the secondary sealing element piston ring. It is suspected that the radial load from the wave spring around the piston ring caused the piston ring to bindup slightly on the housing pilot. Also, the revised piston ring has a larger unbalanced pressure load due to the vent groove relocation. The higher radial pressure load may have caused the piston ring friction drag to exceed the closing forces. The seal was seated by gradually increasing the LOX pressure. Once seated, the seal performance was satisfactory.

The helium seal leakage increased with test time from 0.128 to 0.334 m<sup>3</sup>/minute (4.52 to 11.8 scfm). The leakage stabilized at approximately 0.340 m<sup>3</sup>/minute (12 scfm) after 3 hours and was nearly constant for the last hour. The increase in leakage rate corresponds to the carbon ring wear which was measured posttest. The seal ring average diametral clearance increased from 0.000038 m (0.0014 in.) to 0.00009 m (0.0036 in.). The clearance increased by a factor of 2.4 and the leakage increased by a factor of 2.6. The theoretical leakage is directly proportional to clearance.

It is suspected that the helium seal ring wear occurred due to excessive (0.00007 m; 0.0028 in. TIR) radial runout of the mating ring surface. The irregular lift pad depth could have reduced the lift force so that the seal ring was in rubbing contact until the clearance was equal to the runout.

Build 14 Disassembly. The inspection after 4.16 hours revealed the seals to be in satisfactory condition, except for indication of rubbing contact on the LOX seal and slight wear on the helium seal carbon rings. Both seals were in acceptable condition for continued operation.

The LOX seal carbon face and spiral groove mating ring were slightly scored from rubbing contact. The mating ring surface was smeared with an irregular carbon deposit and indicated slight additional rubbing since the damage which occurred on build 8. The carbon face was scored with light circumferential marks. There was no significant wear on either the mating ring surface or the carbon face.

The LOX seal piston ring was in excellent condition with no visible degradation.

The helium seal carbon rings are worn 0.000043 m (0.0017 in.) diametral on the LOX side and 0.000066 m (0.0026 in.) diametral on the turbine side. The seal ring diametral clearance increased from 0.000036 to 0.000076 m (0.0014 to 0.0030 in.) on the LOX side and from 0.00004 m to 0.0001 m (0.0016 to 0.0042 in.) on the turbine side. The static leakage at 206,843 N/m<sup>2</sup>g (30 psig) purge pressure increased from  $0.043 \text{ m}^3/\text{minute}$  (2650 scim) pretest to  $0.093 \text{ m}^3/\text{minute}$  (5700 scim) posttest. The dynamic leakage during the test series indicates

that the seal ring wear occurred during the first 3 hours with no additional wear during the last hour.

The helium seal lift pads were worn off, except for two which were deeper (0.000031 to 0.000033 m; 0.0012 to 0.0013 in.) when new. The mating ring had a heavy contact pattern with no significant wear.

The seal hardware condition is shown in Fig. 119 through 126.

<u>Results and Conclusions (Tests 187-332 and 666-749).</u> The total test time with liquid oxygen as the sealed fluid was 10 hours and 2.5 minutes for 306 tests. After 4 hours and 28.4 minutes of satisfactory testing on the first build (No. 8), the tester bearings failed and the LOX seal was severely damaged. The test leakage is shown in Fig. 127.

Seal leakage was increasing with considerable scatter during build 8 testing. Seal leakage was more consistent during build 14 testing. At the beginning of four series of tests for build 14, the LOX seal drain pressure redline was exceeded. The probable cause was the seal not seating at the start of a faststart test. With slow pressurization, the seal would seat azd fast-start testing could be continued. The LOX seal was in satisfactory condition following build 14 testing of 4 hours, 9.7 minutes, with slight rubbing on the mate ring and carbon ring.

Rayleigh Pad Floating Ring Helium Seal

<u>Results and Conclusions</u>. A total of 749 tests was performed on five helium seal assemblies for a total of 24 hours of operation. The seal proved to be reliable and long lasting.

Three helium seal assemblies were damaged when bearing failures were experienced; seal performance was satisfactory up to bearing failure.

Static leakage at 206,843  $N/m^2$  (30 psi) versus diametral clearance is shown in Fig. 128.

Wear rate of seal S/N 01, builds 1 through 5 is shown in Fig. 129. Wear rate for S/N 04 builds 9 through 13, is shown in Fig. 130, and wear rate for S/N 05, build 14, is shown in Fig. 131.

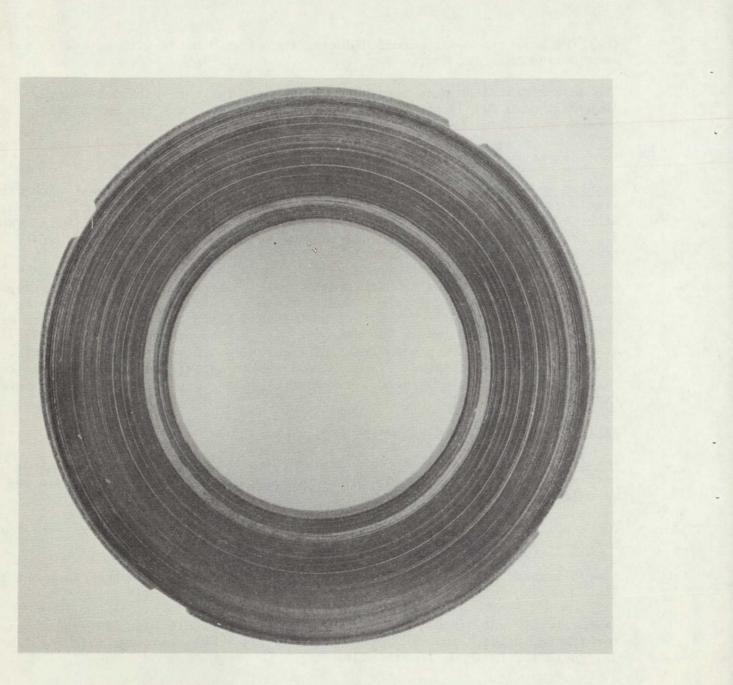


Figure 119. Spiral Groove LOX Seal Carbon Ring (A28-0812-11, S/N 04, Build 14, Posttest 749)



Figure 120. Spiral Groove LOX Seal Mating Ring (C28-0812-10, S/N 03, Build 14, Posttest 749)

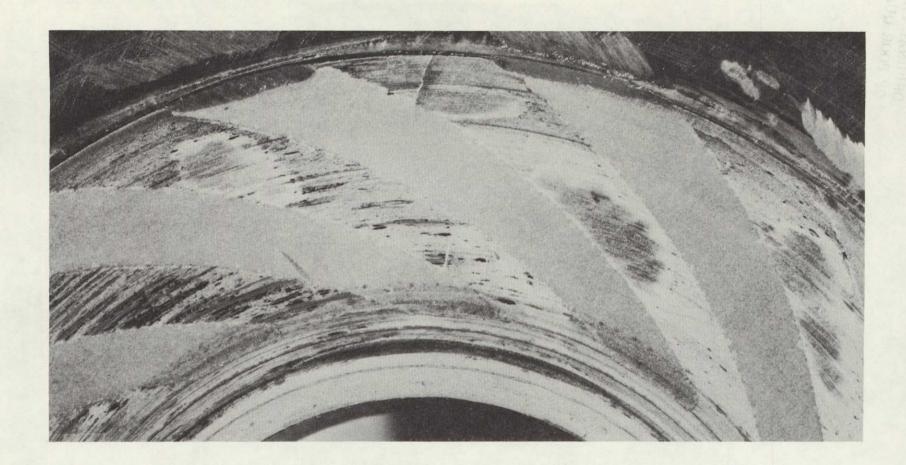


Figure 121. Spiral Groove LOX Seal Mating Ring (C28-0812-10, S/N 03, Build 14, Posttest 749)

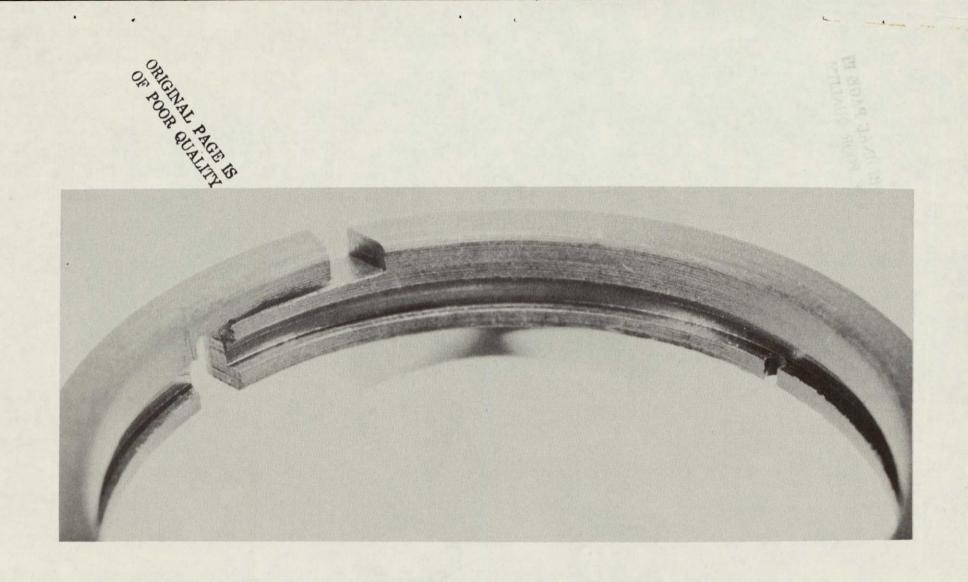


Figure 122. Spiral Groove LOX Seal Piston Ring (A28-0812-12-7456- Rev. C, S/N 03, Build 14, Posttest 749)

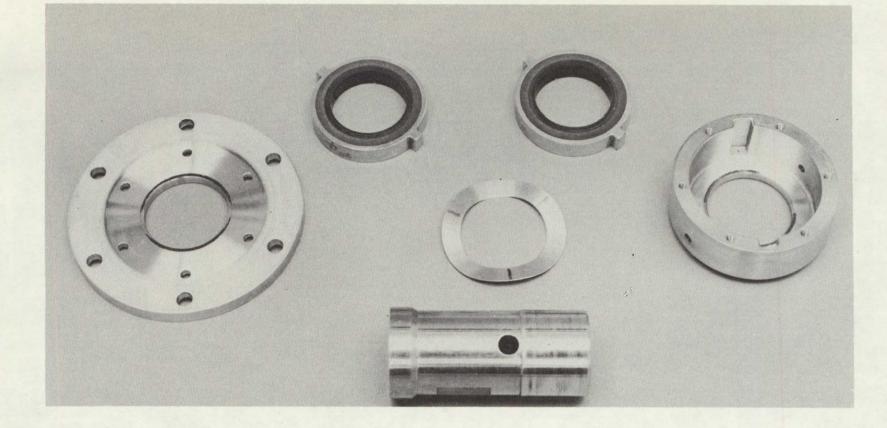


Figure 123. Helium Seal Assembly (RS009690X, S/N 05, Build 14, Posttest 749)

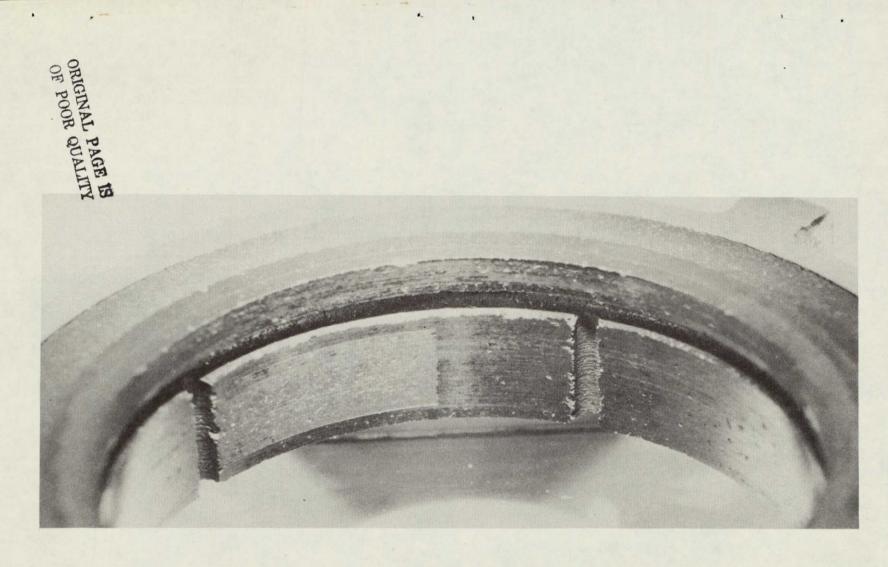


Figure 124. Helium Seal LOX Side Seal Ring (RS009690X, S/N 05, Build 14, Posttest 749)

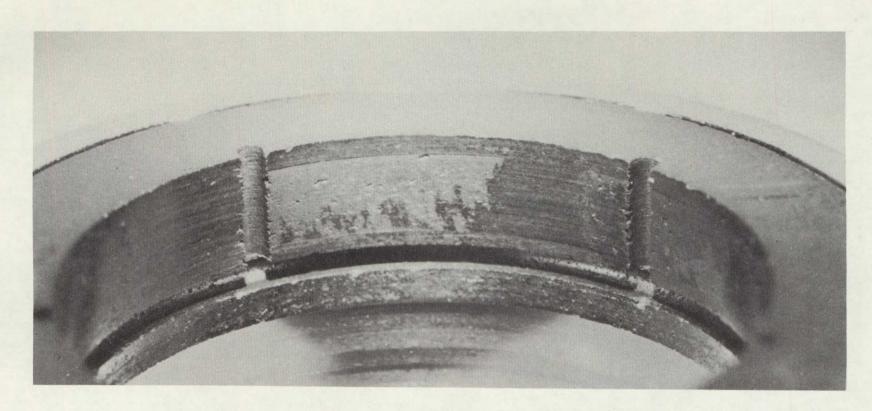
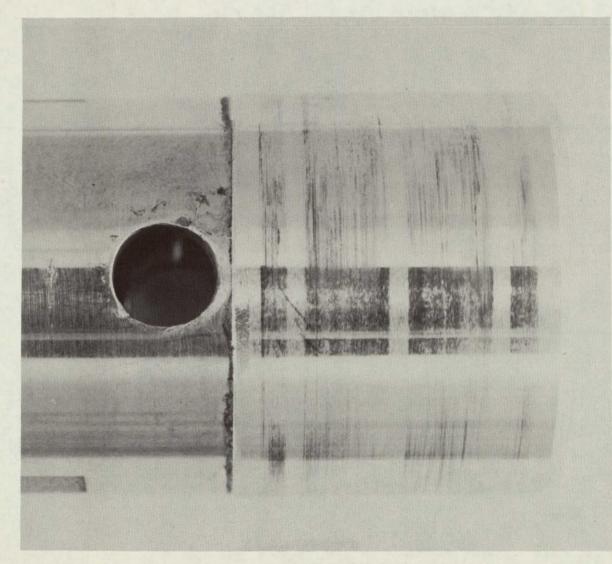


Figure 125. Helium Seal Turbine Side Seal Ring (RS009690X, S/N 05, Build 14, Posttest 749)



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## LOX SIDE

TURBINE SIDE

Figure 126. Helium Seal Mating Ring (RS009667X, S/N 001-1, Build 14, Posttest 749)

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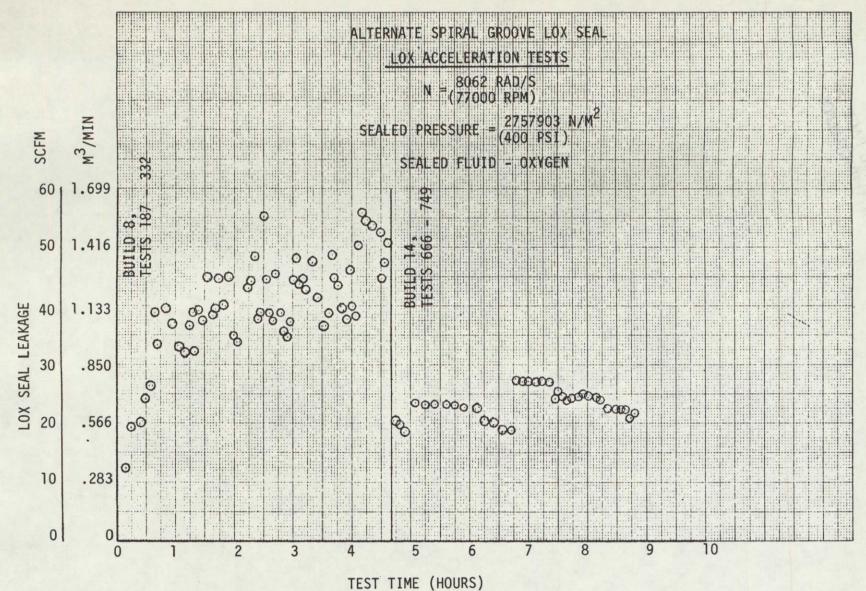


Figure 127. LOX Seal Leakage

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Figure 129. Helium Seal Wear Rate (S/N 01, Builds 1-5)

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	INCH	meters	NASA HELTUM PURGED SEAL
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	.014	3.56	G LOX SLDE
ARANCE	.012	3.05	
DIAMETRAL CLEARANCE	.010	2.54	
DIAM	.008	2.03	
	.006	1.52	
	.004	1.02	
	.002	0.51	
	0	- 0	
			TEST TIME (HOURS)

TEST TIME (HOURS)

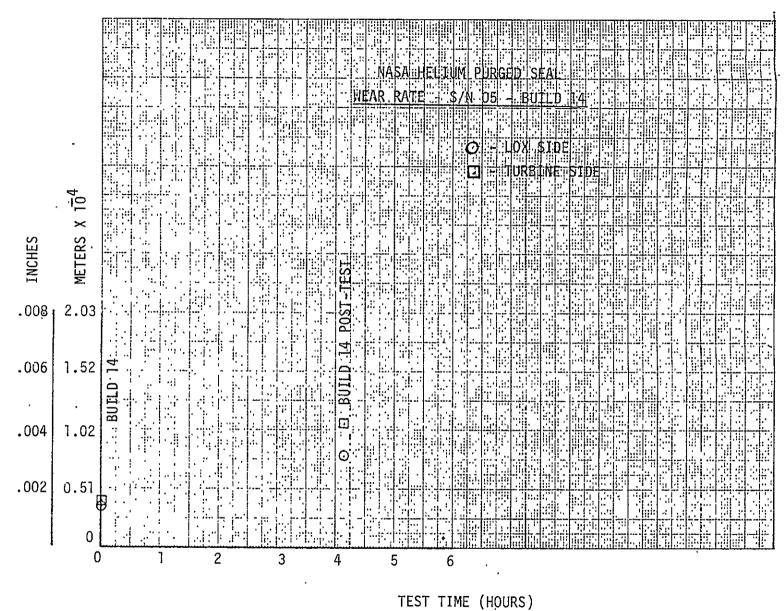
Figure 130. Helium Seal Wear Rate (S/N 04, Builds 9-13)

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DIAMETRAL CLEARANCE



gure 131. Helium Seal Wear Rate (S/N 05, Build 14)

## CONCLUSIONS

1. The shrouded Rayleigh step hydrodynamic lift pad LOX seal is feasible for advanced, small, high-speed oxygen turbopumps.

Total testing time11 hours, 40 minutesTotal starts376Average leakage for 2,413,165 $0.762 \text{ m}^3/\text{minute}$ to 2,771,692 N/m<sup>2</sup>g (350 to $(26.9 \text{ scfm-0}_2)$ 402 psig)LOX seal pressureWear during 10-hour testingNot measurable

- 2. The machined metal bellows secondary LOX seal is not feasible due to carbon wear and difficulty of manufacture.
- 3. The piston ring secondary LOX seal is feasible.

Total testing time	11 hours, 27 minutes
Total starts	365
Condition post testing	Excellent

4. The spiral groove hydrodynamic LOX face seal is feasible.

Total testing time	11 hours, 43 minutes
Total starts	339

During build 8, of 2,757,903 N/m<sup>2</sup>g (400 psig) LOX fast-start tests: Total time 4 hours, 28 minutes Total starts 146 Leakage range 0.544 to 1.580 m<sup>3</sup>/minute

(19.2 to 55.8 scfm) oxygen
with gradual increase with
testing
During build 14, of 2,757,903 N/m<sup>2</sup>g (400 psig) LOX fast-start tests:

Total time 4 hours, 10 minutes Total starts 84 Average leakage 0.657 m<sup>3</sup>/minute (23.2 scfm) oxygen for 64 tests more than 1-minute duration 5. The helium purge intermediate floating ring seal with shrouded Rayleigh step hydrodynamic lift pads on the carbon ring inside diameter is feasible.

Total testing time	23 hours, 23 minutes
Total starts	749
Leakage nange	0.088 to 0.654 m <sup>3</sup> /minute (3.1 to 23.1 scfm) helium

- 6. The small, high-speed seal tester adequately met seal testing requirements. Primary limitation on testing duration was the tester bearing which required inspection after 2.5 hours testing.
- 7. LOX seal Rayleigh step lift pad spring loads were less than the design values for satisfactory testing.
- 8. The reverse pumping element upstream of the LOX face seal was not consistent in dropping the pressure at the seal as a function of speed. Data in Table 13 indicated that the seal upstream cavity pressure was , reduced by 1.6% to 50.9%, at 8062 rad/s (77,000 rpm). The range was probably due to the variation of seal upstream cavity through flow leaking past the slave seal to the bearing cavity. A higher pressure drop could be achieved by closer spacing of the rotating and stationary elements or adding radial vanes to the rotating element.

## REFERENCES

- Zuk, J., P. Ludwig, and R. L. Johnson: <u>Design Study of Shaft Face Seal</u> with Self-Acting Lift Augmentation, I - <u>Self-Acting Pad Geometry</u>, NASA TN D-5744, 1970.
- Zuk, J., P. Ludwig, and R. L. Johnson: <u>Quasi-One-Dimensional Compressible</u> <u>Flow Across Face Seals and Narrow Slots, I - Analysis</u>, NASA TN D-6668, 1972.

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