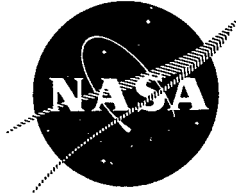


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ELEXIBLE ROTORDYNAMIC ANALYSIS
FINAL REPORT

by

F. A. Shen and E. Mogil

POWER SYSTEMS DIVISION - ROCKETDYNE
A Division of North American Rockwell Corporation

prepared for

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

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by

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A Division of North American Rockwell Corporation
6633 Canoga Avenue
Canoga Park, California

prepared for

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

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CONTRACT NAS 3-13219

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ABSTRACT

A digital computer program was developed to analyze a flexible rotor-bearing system. The program can be used to predict the dynamic performance under various operating conditions. An experimental test program was conducted utilizing a Mark-25 pump rotor, in a vacuum, to generate spin test data to be used for correlation with results of computer simulations of similar rotor unbalance conditions. The mathematical formulations, the computer program verification, the simulation model and test results are presented and discussed.

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SUMMARY

A 10-month study contract, NAS 3-13219, was initiated to update the digital computer program that evolved from a previous study, NAS 3-7996. The work was conducted in three parts. They were 1) update the flexible rotor digital computer program, 2) perform experimental spin tests to generate data, and 3) simulate the experimental tests using the computer program and determine the degree of correlation.

A flexible-rotor dynamic analysis computer program based on Newton's laws of dynamics has been written and computation results verified. The rotor spin and whirl motion were treated as independent but dynamically interacting parameters. Accordingly various whirl-to-spin frequency ratios can be developed as the dynamics relationship dictates. The spin speed of the rotor is controlled by torque applied and combined stiffness, dissipation and inertial loading functions in spin and transverse motion. Both in-phase and out-of-phase stiffness and damping functions are included at all rotor stations. Computation results are of both printout and CRT graph types. After debugging, the computer program was verified in three separate verification steps and the computation results were found to be in good agreement with those from manual and other machine computations. User's instructions are also provided.

The experimental test program, utilizing a Mark-25 pump rotor, was performed to obtain data for correlation with the results of the computer program. The test program consisted of a series of high-speed balancing tests and the eight investigative tests. The investigative tests were simulated with the computer program, and the correlation between theoretical and test results was not very close. Some degree of correlation existed for unbalance conditions at the inducer end of the rotor. Apparently the rotor modeling was not sufficiently descriptive.

The computer program as developed under this study contract can be used as a tool in predicting the dynamic performance of a flexible rotor-bearing system under various operating conditions.

Recommendations are made for further studies to improve the degree of correlation and to increase the computer program capability.

INTRODUCTION

As a continuing effort in advancing the state-of-the-art in rotordynamic analysis, a 10 month program was initiated to develop a computer program to permit more accurate analysis of rotating assemblies, and to apply this program to predictions of the dynamic behavior of the Rocketdyne Mark-25 pump rotor.

The program was accomplished in three tasks: (1) revising and updating the computer program, (2) performing spin tests, and (3) simulation of the experimental data utilizing the computer program.

The mathematical formulation used for the effort reported in this contract, although based on the same general principle as the previous study (Contract NAS3-7996), has been completely rewritten to include the following parameters required by the contract

1. Rotor flexibility due to both shear and bending
2. Damping functions
3. Non-isotropic bearing mounts
4. Nonlinear bearing spring rates
5. Multiple bearing supports

and the following additional items:

6. Linear, speed-sensitive, support bearing stiffness;
7. Enlarged maximum number of rotor-stations to 25, from 12,
8. A displacement vector rotating coordinate system (whirl), as opposed to the spin-speed rotating coordinate used in the 1967 contract; and
9. Up-dating the CRT graphs to include multiple-bearing output data plots.

In addition, spin tests utilizing the Mark-25 pump rotor were performed in the Rocketdyne rotor dynamics facility, for correlation with the

results of the flexible rotor dynamics computer program.

Simulation of the spin test and high speed balancing data was performed with the computer program, utilizing as program inputs the experimentally introduced unbalances.

A checked-out operating computer card deck for the flexible rotor dynamics computer program was delivered to the NASA Project Manager.

This final report presents the results of the research program conducted 4 September 1969 through 1 May 1970. During the course of development of the computer program, certain shortcomings in the program were noted. Conclusions and recommendations are given towards further improving the program computational-to-real time ratio and other rotordynamic parameters.

MATHEMATICAL FORMULATION (THEORY)

In writing the governing equations of a dynamic system, two general approaches may be taken, the energy approach and the momentum approach. The former is generally accomplished by means of Lagrange equations and the latter is carried out according to Newton's Laws of Dynamics. When the traction vs displacement relationship of a dynamic system can be clearly seen, it is faster and more direct to use the momentum approach as it is adopted in this analysis.

The basic assumptions made in the formulation of the general mathematical model are:

1. Representation of a continuous, elastic and distributed mass rotor by a set of discrete masses connected by weightless, elastic shaft sections. The polar and diametral mass moments of inertia and the mass of a rotor section are suitably lumped at the adjacent mass stations. The analysis of the transverse rotordynamic problem is based on a lumped parameter model. With an adequate number of discrete mass stations, the dynamic performance of a rotor can be accurately simulated.
2. Small lateral displacement of a rotor in comparison with the sectional length of a rotor*. This assumption is quite realistic for all rotor dynamics analysis. Under this assumption, the linear moduli of elasticity of the rotor may be used in establishing load and deflection relationship.

* Sectional length of a rotor refers to the length of a rotor section between adjacent discrete mass stations.

3. An axisymmetric rotor geometry and rotor mechanical properties are assumed. While a nonaxisymmetric rotor section will not introduce any solution difficulties, it would result in a very time-consuming computation process. The inclusion of the non-axisymmetric rotor design consideration in a computer analysis is generally not justified as most rotor geometries are or can be considered axisymmetric.

4. The foundation of the rotating machinery is anchored to the secondary frame of reference XYZ in Fig. 1. The secondary frame of reference is permitted to have only translatory motion at a constant velocity with respect to the inertial axes $X_0 Y_0 Z_0$. The surface of the earth may be considered as a secondary inertial frame of reference with a constant velocity relative to the inertial frame of reference $X_0 Y_0 Z_0$. Small angular motion of the secondary frame of reference from the earth rotation will have negligible effects on the dynamics of rotors except those of a gyroscope type of instrument.

Based on the above assumptions, a set of rotordynamics governing equations may be written. Figure 2 depicts a dynamic configuration of a rotor and its casing, and defines certain notations used in the equations. Figure 3 shows that the coordinates X_1 and Y_1 refers to the geometrical center of a rotor section.

EQUATIONS

The following 22 equations constitute the mathematical formulation for the physical model shown in Figure 2:

$$\begin{aligned}
& \ddot{\phi} \sum_i^n \left[I_{\rho_i} + m_i \dot{e}_i^z \right] + \sum_i^n \left\{ m_i e_i \left[\left(\ddot{y}_i + g_y \right) \cos(\phi + \alpha_i) \right. \right. \\
& \quad \left. \left. - \left(\ddot{x}_i + g_x \right) \sin(\phi + \alpha_i) \right] \right\} + \sum_i^n \left[C_{z1i} \dot{\phi}^{C_{z1i}} + C_{z2i} \dot{\phi} \right] \\
& \quad - \left[M_{z1} \dot{\phi}^{M_{z1}} + M_{z2} \dot{\phi} + M_{z3} \right] = 0 \tag{1}
\end{aligned}$$

Eq. (1) defines the torsional moment equilibrium relationship about Z-axis. The first group of terms,

$$\ddot{\phi} \sum_i^n \left[I_{\rho_i} + m_i \dot{e}_i^z \right]$$

represents the inertia torque from the rotor polar mass moments of inertia about the rotor elastic centers. The second group,

$$\sum_i^n \left\{ m_i e_i \left[\left(\ddot{y}_i + g_y \right) \cos(\phi + \alpha_i) - \left(\ddot{x}_i + g_x \right) \sin(\phi + \alpha_i) \right] \right\}$$

denotes the torque derived from the rotor mass inertia forces. The third group

$$\sum_i^n \left[C_{z1i} \dot{\phi}^{C_{z1i}} + C_{z2i} \dot{\phi} \right]$$

is a summation of all rotor spin speed sensitive and constant dissipative torques, and

$$- \left[M_{z1} \dot{\phi}^{M_{z1}} + M_{z2} \dot{\phi} + M_{z3} \right] = 0$$

The last group represents the spin speed sensitive and constant rotor drive torques.

Eqs. (2) and (3) summarize all forces specified at rotor station i along X and Y axis, respectively.

$$\begin{aligned}
 -F_{X_i} = m_i & \left\{ \ddot{X}_i + g_x - e_i \left[\ddot{\phi} \sin(\phi + \alpha_i) + (\dot{\phi})^2 \cos(\phi + \alpha_i) \right] \right\} \\
 & + \left[K_i X_i + C_i \dot{X}_i + K_{pi} Y_i + C_{pi} \dot{Y}_i \right] + \left[K_{HDi} Y_i (\dot{\phi} - K_{FL} \omega_i) \right. \\
 & \left. + C_{HDi} \dot{Y}_i (\dot{\phi} - C_{FL} \omega_i) \right] + A_i \left\{ \left[(\dot{\phi} - \dot{\phi}_0) \right] \left[N_{Bik} \right. \right. \\
 & \left. \left. + B_{Bik} \left(\sqrt{X_{Bi}^2 + Y_{Bi}^2} - \rho_{Bik} \right) \right] + K_{Bik} \right\} \left[\left(X_{Bi}^2 + Y_{Bi}^2 \right)^{\frac{H_{Bik}-1}{2}} \right. \\
 & \left. + D_{Bik} + \frac{E_{Bik}}{\sqrt{X_{Bi}^2 + Y_{Bi}^2}} \right] X_{Bi} + C_{Bxi} \dot{X}_i \left. \right\} \quad (2)
 \end{aligned}$$

$$\begin{aligned}
 -F_{Y_i} = m_i & \left\{ \ddot{Y}_i + g_y + e_i \left[\ddot{\phi} \cos(\phi + \alpha_i) - (\dot{\phi})^2 \sin(\phi + \alpha_i) \right] \right\} \\
 & + \left[K_i Y_i + C_i \dot{Y}_i - K_{pi} X_i - C_{pi} \dot{X}_i - K_{HDi} X_i (\dot{\phi} - K_{FL} \omega_i) \right. \\
 & \left. - C_{HDi} \dot{X}_i (\dot{\phi} - C_{FL} \omega) \right] + A_i \left\{ \left[(\dot{\phi} - \dot{\phi}_0) \right] \left[N_{Bik} \right. \right. \\
 & \left. \left. + B_{Bik} \left(\sqrt{X_{Bi}^2 + Y_{Bi}^2} - \rho_{Bik} \right) \right] + K_{Bik} \right\} \left[\left(X_{Bi}^2 + Y_{Bi}^2 \right)^{\frac{H_{Bik}-1}{2}} \right. \\
 & \left. + D_{Bik} + \frac{E_{Bik}}{\sqrt{X_{Bi}^2 + Y_{Bi}^2}} \right] Y_{Bi} + C_{BYi} \dot{Y}_i \left. \right\} \quad (3)
 \end{aligned}$$

The first group in Eq. (2) and similarly with Eq. (3).

$$m_i \left\{ \ddot{X}_i + g_x - e_i \left[\ddot{\phi} \sin(\phi + \alpha_i) + (\dot{\phi})^2 \cos(\phi + \alpha_i) \right] \right\}$$

defines the inertia and gravity and/or G-loading forces along X-axis. The second group,

$$\left[K_i X_i + C_i \dot{X}_i + K_{pi} Y_i + C_{pi} \dot{Y}_i \right]$$

denotes the rotor surface forces at a rotor station.

$$K_i X_i, C_i \dot{X}_i, K_{pi} Y_i \text{ and } C_{pi} \dot{Y}_i$$

are in-phase stiffness force, in-phase damping force, out-of-phase stiffness force and out-of-phase damping force respectively. The in-phase force is in line but opposite to the displacement or velocity vector and the out-of-phase force leads 90° from the displacement or velocity vector. The third group

$$\left[K_{HDi} Y_i (\dot{\phi} - K_{Fi} \omega_i) + C_{HDi} \dot{Y}_i (\dot{\phi} - C_{Fi} \omega_i) \right]$$

represents the rotor surface forces at a rotor station. They are the combined rotor-spin and whirl speed sensitive out-of-phase stiffness and damping forces. These forces may be generated in a non-synchronous whirl motion such as that encountered in a fluid-film bearing or similar components. The last group,

$$A_i \left\{ \left[(\dot{\phi} - \dot{\phi}_0) \right] \left[N_{Bik} + B_{Bik} \left(\sqrt{X_{Bi}^2 + Y_{Bi}^2} - \rho_{Bik} \right) \right] + K_{Bik} \right\}$$

$$\left[\left(\sqrt{X_{Bi}^2 + Y_{Bi}^2} \right)^{\frac{1}{2} B_{Bik} - 1} + D_{Bik} + \frac{E_{Bik}}{\sqrt{X_{Bi}^2 + Y_{Bi}^2}} \right] X_{Bi} + C_{Bik} \dot{X}_i \left\}$$

denotes the nonlinear bearing forces at a bearing station. $\dot{\phi}_0$ is a rotor spin speed constant and $\dot{\phi}$ is the actual rotor spin speed. $(\dot{\phi} - \dot{\phi}_0) N_{BiK}$ is used to simulate the spin speed sensitive bearing stiffness at zero journal displacement. B_{BiK} is the spin speed and journal displacement sensitive stiffness coefficient. X_{Bi} and Y_{Bi} are the journal displacement coordinates and ρ_{BiK} is the lower bearing displacement limit for K th bearing stiffness section of bearing i. K_{BiK} is the journal displacement sensitive bearing stiffness.

$$K_{BiK} \left[\left(X_{Bi}^2 + Y_{Bi}^2 \right)^{\frac{H_{BiK}-1}{2}} + D_{BiK} + \frac{E_{BiK}}{\sqrt{X_{Bi}^2 + Y_{Bi}^2}} \right] X_{Bi}$$

is the nonlinear bearing stiffness force function, and $C_{BXi} \dot{x}_i$ the X-component damping force at the bearing.

Eqs. (4) and (5) describe all the specified moments defined about Y and X axis respectively.

$$\begin{aligned} -\left(l_{i-1} + l_i \right) M_{Yi} = & I_{Di} \left[\ddot{x}_{i+1} - \ddot{x}_{i-1} - \beta_i (l_{i-1} + l_i) \left(\dot{\phi} \sin(\phi + \gamma_i) \right. \right. \\ & \left. \left. + (\dot{\phi})^2 \cos(\phi + \gamma_i) \right) \right] \\ & + I_{\rho i} \dot{\phi} \left[(\dot{y}_{i+1} - \dot{y}_{i-1}) + \beta_i (l_{i-1} + l_i) \dot{\phi} \cos(\phi + \gamma_i) \right] \\ & + K_{\phi i} (x_{i+1} - x_{i-1}) + C_{\phi i} (\dot{x}_{i+1} - \dot{x}_{i-1}) \\ & + K_{\rho i} (y_{i+1} - y_{i-1}) + C_{\rho i} (\dot{y}_{i+1} - \dot{y}_{i-1}) \\ & + K_{\phi Di} (\dot{\phi} - K_{\phi i} \omega_i) (y_{i+1} - y_{i-1}) + C_{\phi Di} (\dot{\phi} - C_{\phi i} \omega_i) (\dot{y}_{i+1} - \dot{y}_{i-1}) \end{aligned} \quad (4)$$

$$\begin{aligned}
(l_{i+1} + l_i) M_{x_i} = & I_{Di} \left[\left(\ddot{Y}_{i+1} - \ddot{Y}_{i-1} \right) + \beta_i (l_{i+1} + l_i) \left[\ddot{\phi} \cos(\phi + \gamma_i) \right. \right. \\
& \left. \left. - (\dot{\phi})^2 \sin(\phi + \gamma_i) \right] \right] \\
& - I_{\rho i} \dot{\phi} \left[\left(\dot{X}_{i+1} - \dot{X}_{i-1} \right) - \beta_i (l_{i+1} + l_i) \dot{\phi} \sin(\phi + \gamma_i) \right] \\
& + K_{\phi i} (Y_{i+1} - Y_{i-1}) + C_{\phi i} (\dot{Y}_{i+1} - \dot{Y}_{i-1}) \\
& - K_{\rho i} (X_{i+1} - X_{i-1}) - C_{\rho i} (\dot{X}_{i+1} - \dot{X}_{i-1}) \\
& - K_{\phi HD i} (\dot{\phi} - C_{\phi F i} \omega_i) (X_{i+1} - X_{i-1}) - C_{\phi HD i} (\dot{\phi} - C_{\phi F i} \omega_i) (\dot{X}_{i+1} - \dot{X}_{i-1})
\end{aligned} \tag{5}$$

The first group in Eq. (4), and similarly with Eq. (5)

$$I_{Di} \left[\ddot{x}_{i+1} - \ddot{x}_{i-1} - \beta_i (l_{i+1} + l_i) \left(\ddot{\phi} \sin(\phi + \gamma_i) + (\dot{\phi})^2 \cos(\phi + \gamma_i) \right) \right]$$

denotes the inertial moment resulting from the transverse mass moment of inertia at a rotor station. In calculating the bending moments at a rotor station the slopes and their time derivatives are approximated by that of the chord joining the rotor centers at the adjacent stations. Using the chord approximations of the slopes the number of unknowns involved in the solution of the rotordynamics equations was substantially reduced. Two additional unknown variables at each station, i.e., the force and moment slope coefficients would otherwise have to be included in the rotordynamics equations. β_i and γ_i are the initial misalignment and angular orientation of the axis of the mass moments of inertia with respect to rotor elastic center line and initial rotor spin angular position respectively.

The second group,

$$I_{\rho i} \ddot{\phi} \left[\left(\dot{Y}_{i+1} - \dot{Y}_{i-1} \right) + S_i \left(l_{i-1} + l_i \right) \dot{\phi} \cos \left(\phi + \gamma_i \right) \right]$$

represents the inertia moment from the polar mass moment of inertia at a rotor station.

The third group

$$K_{\phi i} \left(X_{i+1} - X_{i-1} \right) + C_{\phi i} \left(\dot{X}_{i+1} - \dot{X}_{i-1} \right)$$

denotes the rotor surface forces and they are the in-phase stiffness and damping moment due to a rotor misalignment motions with respect to the rotor casing or bearing at a rotor station.

The fourth group,

$$K_{\phi Pi} \left(Y_{i+1} - Y_{i-1} \right) + C_{\phi Pi} \left(\dot{Y}_{i+1} - \dot{Y}_{i-1} \right)$$

similarly represents the rotor surface moments consisting of the out-of-phase stiffness and damping moment from rotor misalignment motions.

The last group,

$$K_{\phi HDi} \left(\dot{\phi} - K_{\phi Fi} \omega \right) \left(Y_{i+1} - Y_{i-1} \right) + C_{\phi HDi} \left(\dot{\phi} - C_{\phi Fi} \omega \right) \left(\dot{Y}_{i+1} - \dot{Y}_{i-1} \right)$$

denotes the rotor surface moments. They are the out-of-phase, rotor spin and whirl sensitive stiffness and damping moments, similar to the third group in Eq. (2) or (3).

Eqs. (6) and (7) define the force and moment and equilibrium relationships in XZ and YZ planes. This is where the force and moment influence coefficients e_{ij} and b_{ij} are used to solve the basic rotordynamics equations

for every rotor station except that for the first and last non-linear bearings. The basic rotordynamics equations for the first and last bearing stations are Eqs. (8) through (11). To solve the system of rotordynamics equations, first convert into the rotating coordinates and then substitute F_{xi} , F_{yi} , M_{yi} and M_{xi} functions in Eqs. (2) through (5) into Eqs. (6) through (11). The values of c_{ij} and b_{ij} used are computed in subroutine "INFLCØ."

Eqs. (1) and (6) through (11) are solved simultaneously for the acceleration terms by means of ISIMDD subroutine. Using the current values of the acceleration and velocity terms, integration over a time interval will be made to obtain the solution for a rotor motion corresponding to a new real time. The process repeats for each time interval until a desired total time period for rotor motion is covered.

The influence coefficients are calculated by finding the rotor deflections (in.) at various stations when a unit transverse force (1 lb) or unit transverse moment (1 in-lb) is applied at a station. Rotor deflections considering rotor shear and bending elasticity are separately computed and then combined. The influence coefficients are based on the rotor-configuration supported on two knife-edge bearings located at the first and last support bearings. The rotor dynamic relationship at these two bearings are defined by Eqs. (8) through (11). Sample results in force and moment influence coefficients are shown in Appendix A. The meaning of row and column numbers in relation to the definitions of the influence coefficients is explained.

$$\sum_i^n \left[F_{xi} C_{ij} + M_{yi} b_{ij} \right] + X_{bl} + \left(X_{bz} - X_{bl} \right) \frac{Z_j - S}{L} - X_j = 0 \quad (6)$$

$$\sum_i^n \left[F_{yi} C_{ij} - M_{xi} b_{ij} \right] + Y_{bl} + \left(Y_{bz} - Y_{bl} \right) \frac{Z_j - S}{L} - Y_j = 0 \quad (7)$$

Eqs. (8), (9), (10) and (11) denote the moment and force equilibrium relationships for the first and last nonlinear bearing station in X-Z and Y-Z plane, respectively. These equations are solved simultaneously with Eqs. (6) and (7).

$$\sum_i^n \left[(Q - Z_i) F_{x_i} - M_{y_i} \right] = 0 \quad (8)$$

$$\sum_i^n F_{x_i} = 0 \quad (9)$$

$$\sum_i^n \left[(Q - Z_i) F_{y_i} + M_{x_i} \right] = 0 \quad (10)$$

$$\sum_i^n F_{y_i} = 0 \quad (11)$$

Eqs. (12) and (14) denote the force equilibrium relationships at the bearing-and-mount station along X and Y axis, respectively. The bearing and mount damping forces are not included. Since zero bearing mass assumption is used, the stiffness force in the bearing should only equal that in the mount and similar relationship exists for the damping forces.

$$A_i \left\{ \left[\dot{\phi} - \dot{\phi}_0 \right] \left[N_{Bik} + B_{Bik} (\rho_{BX_i} - \rho_{Bik}) \right] + K_{Bik} \right\} \left(\rho_{BX_i}^{(H_{Bik}-1)} + D_{Bik} + \frac{E_{Bik}}{\rho_{BX_i}} \right) X_{Bi} - K_{MX_i} [X_i - X_{B_i}] \left. \right\} = 0 \quad (12)$$

Eqs. (13) and (15) are expressions for ρ_{BX_i} and ρ_{BY_i} used in Eqs. (12) and (14) respectively.

$$\rho_{BX_i} = \sqrt{X_{Bi}^2 + \left[\frac{Y_i}{\frac{K_{MX_i}}{K_{MY_i}} \left(\frac{X_i}{X_{Bi}} - 1 \right) + 1} \right]^2} \quad (13)$$

$$A_i \left\{ \left[\dot{\phi} - \dot{\phi}_0 \right] \left[N_{Bik} + B_{Bik} (\rho_{BY_i} - \rho_{Bik}) \right] + K_{Bik} \right\} \left(\rho_{BY_i}^{(H_{Bik}-1)} + D_{Bik} + \frac{E_{Bik}}{\rho_{BY_i}} \right) Y_{Bi} - K_{MY_i} [Y_i - Y_{B_i}] \left. \right\} = 0 \quad (14)$$

$$\rho_{BY_i} = \sqrt{Y_{Bi}^2 + \left[\frac{X_i}{\frac{K_{MY_i}}{K_{MX_i}} \left(\frac{Y_i}{Y_{Bi}} - 1 \right) + 1} \right]^2} \quad (15)$$

Eqs. (12) through (15) establish the nonlinear bearing and mount stiffness forces along X and Y axes. The purpose of these equations is to solve for the nonlinear bearing journal displacements through an iterative process, a faster method than the closed form solution.

Eqs. (16) through (21) define the various coordinates used in the basic rotordynamics equations. Under the zero bearing mass assumption, the bearing damping can be expressed in terms of the combined bearing-and-mount damping forces. The relationships are shown in Eqs (16) through (19).

$$\dot{X}_{Bi} = \frac{C_{BXL} \dot{X}_i}{C_{Bi}} \quad (16)$$

$$\dot{Y}_{Bi} = \frac{C_{BYL} \dot{Y}_i}{C_{Bi}} \quad (17)$$

$$\dot{X}_{Mi} = \frac{C_{BXL} \dot{X}_i}{C_{MXi}} \quad (18)$$

$$\dot{Y}_{Mi} = \frac{C_{BYL} \dot{Y}_i}{C_{MYi}} \quad (19)$$

Also based on zero-bearing-mass assumption the journal and mount displacement are collinear as shown in Eqs. (20) and (21).

$$X_{mi} = X_i - X_{Bi} \quad (20)$$

$$Y_{mi} = Y_i - Y_{Bi} \quad (21)$$

Eq. (22) defines the whirl speed of shaft center at station i .

$$\omega_i = \frac{\dot{Y}_i X_i - \dot{X}_i Y_i}{X_i^2 + Y_i^2} \quad (22)$$

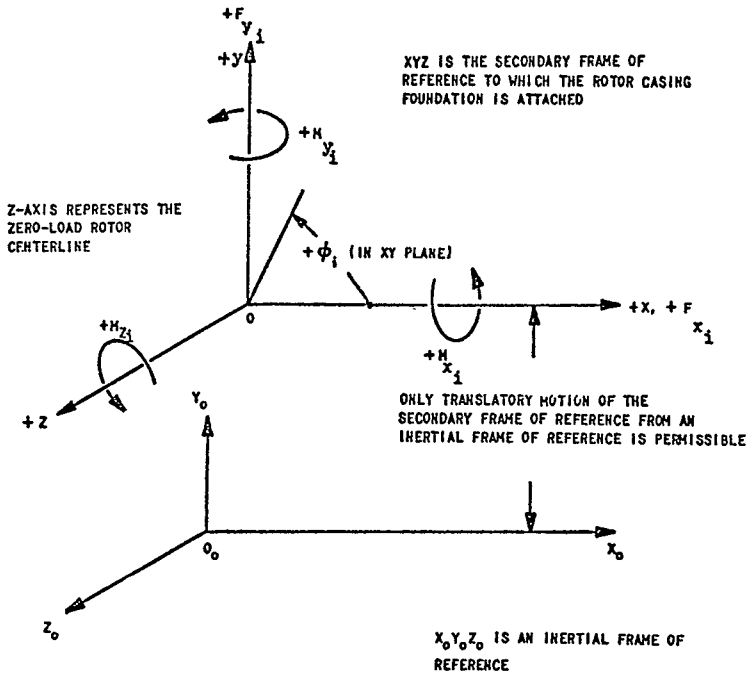
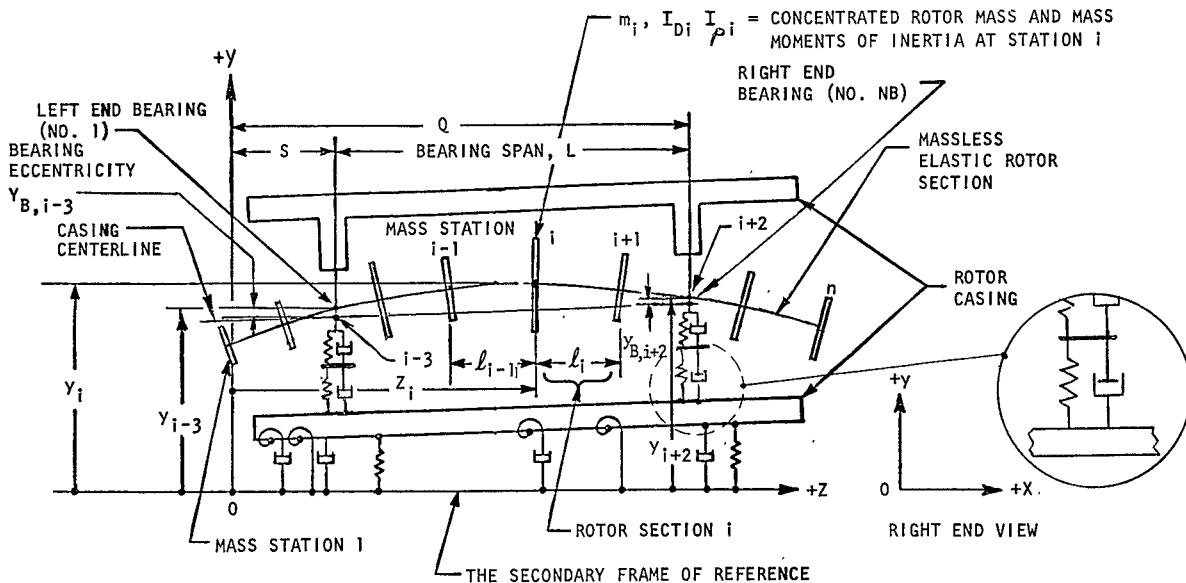


Figure 1 - Relation Between the Secondary Frame of Reference and an Inertial Frame of Reference



- NOTE: 1. x_i AND $x_{B,i}$ ARE NOT SHOWN IN THE SKETCH
2. THE COORDINATES x_i AND y_i DO NOT NECESSARILY REPRESENT THE PHYSICAL DISTANCES AS SHOWN. THEY SHOULD BE INTERPRETED AS THE DISPLACEMENT FROM THEIR CORRESPONDING ZERO-LOAD POSITIONS.
3. IN THE PRESENT ANALYSIS, THE CASING IS ASSUMED TO BE RIGIDLY ATTACHED TO THE SECONDARY FRAME OF REFERENCE.

Figure 2. Schematic of the General Physical Model

The rotating coordinate system (Fig. 3) is used to reduce the integration time as it provides lower acceleration magnitude compared with that from conventional stationary coordinate system. The origin of the rotating coordinates coincides with the static rotor center at a station.

The static rotor center corresponds to that caused by g_x and g_y for the rotor with zero spin speed. The rotating coordinates used are synchronous with the rotor displacement vector at that rotor station. Let the static deflections at a rotor station be ΔX_i and ΔY_i ; and ρ_i and Θ_i be the displacement vector and angular displacement of the rotating coordinates respectively. Then

$$\begin{aligned} X_i &= \rho_i \cos \theta_i + \Delta X_i \\ Y_i &= \rho_i \sin \theta_i + \Delta Y_i \end{aligned}$$

Differentiate once and twice with respect to time and using appropriate substitution, one obtains,

$$\begin{aligned} \dot{X}_i &= \dot{\rho}_i \cos \theta_i - \rho_i \dot{\theta}_i \sin \theta_i \\ \dot{Y}_i &= \dot{\rho}_i \sin \theta_i + \rho_i \dot{\theta}_i \cos \theta_i \\ \ddot{X}_i &= -\dot{\theta}_i \dot{Y}_i - \dot{\rho}_i \dot{\theta}_i \sin \theta_i + \ddot{\rho}_i \cos \theta_i - \ddot{\theta}_i \rho_i \sin \theta_i \\ \ddot{Y}_i &= \dot{\theta}_i \dot{X}_i + \dot{\rho}_i \dot{\theta}_i \cos \theta_i + \ddot{\rho}_i \sin \theta_i + \ddot{\theta}_i \rho_i \cos \theta_i \end{aligned}$$

The variables X_i , Y_i , and their time derivatives in the rotordynamic equations are then replaced with the equivalent rotating coordinates prior to the solution of the rotordynamics equations.

NOMENCLATURE

- A_i = bearing location notation, $A_i = 1$ implies a bearing at station i ; $A_i = 0$, no bearing at station i dimensionless
- b_{ij}, C_{ij} = moment and force influence coefficient, in/(in-lb) and in/lb, respectively
- B_{Bik} = rotative speed sensitive bearing stiffness displacement coefficient for k th stiffness section of bearing i ; for dimension, see Note 2
- C_{Bi} = bearing damping coefficient for \dot{X}_{Bi} and \dot{Y}_{Bi} , (lb-sec)/in
- C_{BXi}, C_{BYi} = equivalent bearing-and-mount damping coefficients for \dot{X}_i, \dot{Y}_i , respectively, (lb-sec)/in
- C_{Fi} = whirl frequency factor for damping force coefficient, dimensionless
- C_{HDi} = out-of-phase whirl-frequency sensitive damping force coefficient, (lb-sec²)/(in-radian)
- $C_i, C_{\phi i}$ = in-phase damping force and moment coefficient due to relative motion between rotor and casing (lb-sec)/in and (in-lb-sec)/rad, respectively
- C_{MXi}, C_{MYi} = non-isotropic mount damping coefficient for $\dot{X}_{Mi}, \dot{Y}_{Mi}$, respectively, (lb-sec)/in
- $C_{Pi}, C_{\phi Pi}$ = out-of-phase damping force and moment coefficient due to relative motion between rotor and casing (lb-sec)/in and (in-lb-sec)/rad, respectively
- C_{Zi} = torsional friction exponent, Eq. (1), dimensionless

- C_{Z1i}, C_{Z2i} = torsional friction coefficients, Eq. (1),
dimension of $C_{Z1i} \dot{\phi}^{CZi}$ or $C_{Z2i} \dot{\phi}$ in in-lb
- $C_{\phi i}$ = damping moment coefficient, (lb-in-sec)/rad.
- $C_{\phi Fi}$ = whirl-frequency factor for damping moment coefficient,
dimensionless
- $C_{\phi HDi}$ = out-of-phase, whirl frequency sensitive, damping moment
coefficient, (lb-in-sec²)/radian²
- D_{BiK}, E_{BiK} = nonlinear bearing stiffness coefficients for kth stiffness
section of bearing i. For dimension, see Note 1.
- e_i = mass eccentricity at rotor station i, in
- F_{xi}, F_{yi} = forces in X and Y direction, lb
- g_x, g_y = gravity or g-loading in X and Y direction, in/sec²
- H_{BiK} = nonlinear bearing stiffness exponent for kth stiffness
section of bearing i. For dimension, see Note 1
- $I_{Di}, I_{\rho i}$ = rotor diametral and polar mass moments of inertia at
station i, lb-in-sec², respectively
- K_{BiK} = rotative-speed sensitive bearing stiffness coefficient for
Kth stiffness section of bearing i, lb/in
- $K_{\phi i}$ = whirl frequency factor for stiffness force coefficient,
dimensionless
- $K_i, K_{\phi i}$ = in-phase stiffness force and moment coefficient due to
relative motion between rotor and casing at station i,
lb/in and (in-lb)/rad, respectively.

- $K_{HDi}, K_{\dot{\phi}HDi}$ = out-of-phase hydrodynamic force and moment coefficient of balance piston, (lb-sec)/(in-rad) and (lb-in-sec)/rad², respectively
- K_{MXi}, K_{MYi} = non-isotropic stiffness coefficients for X_{Mi}, Y_{Mi} , respectively, lb/in
- $K_{pi}, K_{\dot{\phi}pi}$ = out-of-phase stiffness force and moment coefficient due to the relative motion between rotor and casing at station i, lb/in and in-lb/rad, respectively
- $K_{\dot{\phi}pi}$ = whirl frequency factor for stiffness moment coefficient, dimensionless
- l_i = rotor section length between the adjacent mass stations i and i + 1, in
- L = length between bearings No. 1 and NB, in
- m_i = rotor mass at station i, (lb-sec²)/in
- M_{xi}, M_{yi}, M_{zi} = moments about X, Y, and Z axes, respectively, in/lb
- M_Z = exponent for speed sensitive rotor drive torque coefficient
- M_{Z1}, M_{Z2}, M_{Z3} = coefficients for rotor drive torque, dimensions of $M_{Z1}\dot{\phi}^{M_Z}$, $M_{Z2}\dot{\phi}$, or M_{Z3} are in-lb
- n = total number of rotor discrete mass stations, dimensionless
- N_{Bik} = rotative-speed sensitive bearing stiffness coefficient for Kth stiffness section of bearing i. For dimension, see Note 2.
- NB = The last bearing number, or total number of non-linear stiffness bearings.
- \hat{Q} = length between No. NB bearing and mass station 1, in

- S = length between No. 1 bearing and the mass station 1, in
 t = real time, sec
 $X_{Bi}, Y_{Bi}, \dot{X}_{Bi}, \dot{Y}_{Bi}$ = bearing displacement and velocity coordinates, in and in/sec, respectively
 $X_{Mi}, Y_{Mi}, \dot{X}_{Mi}, \dot{Y}_{Mi}$ = mount displacement and velocity coordinates, in and in/sec, respectively
 X_i, Y_i = displacement of the rotor geometrical center from its zero-load position in X and Y direction, respectively, at station i, in
XYZ = the secondary frame of reference to which the rotor casing is attached (Fig. 3)
 X_o, Y_o, Z_o = an inertial frame of reference (Fig. 1)
 Z_i = Z coordinate of ith mass from first mass, in
 α_i = orientation of the ith mass eccentricity vector from that of the first rotor mass measured in the direction of rotation, rad
 ω_i = angular whirl velocity of the rotor geometric center at station i, rad/sec
 $\dot{\phi}_o$ = a spin speed sensitive bearing stiffness parameter, rad/sec as defined in Eq. (2), (3), (12) and (14)
 ϕ = spin speed angular displacement of the torsionally rigid rotor, measured from the positive X-axis, rad

NOTE 1: Group dimension of $K_{Bik} \left[\frac{H_{Bik}-1}{(x_{Bi}^2 + y_{Bi}^2)^2} + D_{Bik} + \frac{E_{Bik}}{x_{Bi}^2 + y_{Bi}^2} \right]$
 is lb/in.

NOTE 2: Group dimension of $(\dot{\phi} - \phi_0) N_{Bik} + B_{Bik} (\sqrt{x_{Bi}^2 + y_{Bi}^2} - \rho_{Bik})$
 is the same as that of K_{Bik} .

ρ_i = initial misalignment between the axis of the mass moments
 of inertia and the elastic axis of rotor section i, radians

γ_i = angular position of the X-Y plane projection of the axis
 of ith mass moments of inertia measured from that of the
 first, radians

ρ_{Bik} = lower bearing displacement limit for kth bearing stiffness
 section of bearing i, in

ρ_{BXi}, ρ_{BYi} = as defined by Eqs. (13) and (15), respectively, in.

Subscripts

i = pertinent to the ith station or ith rotor section

b1, b2 = pertinent to first and last nonlinear bearing
 stations respectively

Time Derivative Notation

$\dot{x}, \ddot{x} \dots = \frac{dx}{dt}, \frac{d^2x}{dt^2} \dots$ respectively, in/sec, in/sec²

$\dot{\phi}, \ddot{\phi} = \frac{d\phi}{dt}, \frac{d^2\phi}{dt^2}$, respectively, rad/sec, rad/sec²

VERIFICATION OF THE COMPUTER PROGRAM

After debugging the computer program three stages of verification of the program were made. The detailed description of each stage of verification follows:

Verification of the Basic Mathematical Formulation

As a preliminary checkout of the "Fortran" coding of the mathematical formulation, a 5-station flexible rotor-bearing configuration, operating under a steady-state condition was chosen. Hand computed, initial rotor motion conditions simulating a steady-state operation were used. The hand calculation was based on the equilibrium between the centrifugal loading and bearing restoring forces. The computer result in rotating coordinates indicated a very small residual acceleration on the order of 10^{-9} of the magnitude of the centrifugal acceleration.

The residual net acceleration, not being zero, is due to input data not having a sufficient number of significant figures and computation round-off errors. The comparative results are shown in Table 1.

Verification of the Computation Results

The computation results of a 17-station rotor model operating under a steady-state condition were compared with that from an existing rotor-dynamic response program based on a matrix iteration technique and they were found in good agreement within a maximum of 1.5% deviation for the case studied. Two different approaches in mathematical formulations are used in these two types of computer programs. The accuracy of the flexible rotor dynamic computer program is considered to be adequate. Table 2 lists the comparative results.

Evaluation of the Accuracy and Convergence of the Computation Process

Using a 5-station rotor test model with a spin speed of 95.5 rpm, a computation with many integration steps was performed. A reasonably large

TABLE 1

5-STATION ROTOR-BEARING CONFIGURATION CHECKOUT
COMPARATIVE RESULTS

Station No.	1	2	3	4	5
<u>DISPLACEMENT VECTORS</u>					
Manually Calculated	.0017355"	.0009999"	.00077360"	.0009999"	.0017355"
Computer Program	.0017356"	.001"	.00077365"	.001"	.0017356"
<u>COMPUTER*CALCULATED ACCELERATION IN ROTATING COORDINATES</u>					
Radial	-3.716×10^{-9} in/sec	3.440×10^{-9}	$.1986 \times 10^{-9}$	3.440×10^{-9}	-3.716×10^{-9}
Tangential	8.116×10^{-16}	-1.970×10^{-14}	1.315×10^{-14}	-2.503×10^{-14}	8.116×10^{-16}

* Predicted Acceleration = zero.

TABLE 2

COMPARISON OF MARK 25, 17 MASS MODEL
NEW PROGRAM VS OLD PROGRAM

Station	Deflection at Station $\times 10^{-5}$		% Difference
	New Transient Program	Old Steady-State Response Program	
1 (Inducer) End	.3554	.3548	.17
2	.3145	.3140	.16
3	.2028	.2023	.25
4	.1419	.1415	.28
5	.0845	.08414	.43
6	.04133	.04098	.85
7	.02310	.02276	1.50
8	.05076	.05099	.45
9	.1377	.1377	0
10	.1908	.1907	.05
11	.2364	.2363	.04
12	.3302	.3300	.06
13	.3517	.3514	.09
14	.3963	.3960	.08
15	.4159	.4156	.07
16	.5093	.5090	.06
17	.9782	.9784	.02

integration tolerance of 10% was used. The result in rotor displacement appears to stay close to the exact value. The instantaneous whirl to spin frequency ratio indicated a substantial oscillation at times, although the trend of convergence appeared definite. With a smaller integration tolerance as is usual (less than 10%), the magnitude of oscillations should be reduced accordingly. The integrated whirl displacement versus time function (not shown here) was reasonably smooth. Two graphs showing the variation of a deflection vector and whirl-to-spin frequency ratio are attached for reference (Figs. 4 and 5).

As evident from Fig. 4 and 5, the instantaneous whirl to spin frequency ratio appears to be coupled to the rotor displacement motion. The energy variation in the displacement motion is complemented by that in whirl motion through the mechanism similar to that of Coriolis acceleration. Because of the radius of whirl motion being rather small the angular velocity change is greatly magnified for a small change in rotary kinetic energy. Since the computer program for this rotor configuration was not run for a sufficient time period, it cannot be certain whether the relatively large oscillations between 0.12 and 0.14 second of real time were manifestation of numerical instability. It, however, does not appear likely to be the case.

The minimum DT (Delta Time) to avoid instability depends on the rotor mass-to-stiffness parameter ratio and no definite such relationships have yet been established.

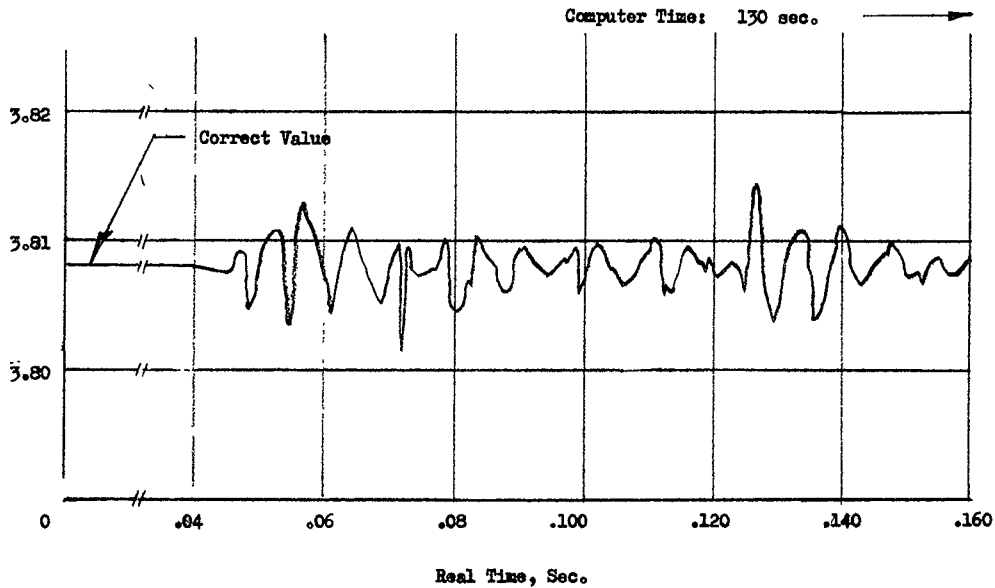
Rotor Displacement at Station #3, Inches $\times 10^{-5}$ 

Figure 4 Data from a 5-Mass 95.5 rpm Flexible Rotor
in a Steady-State Operation

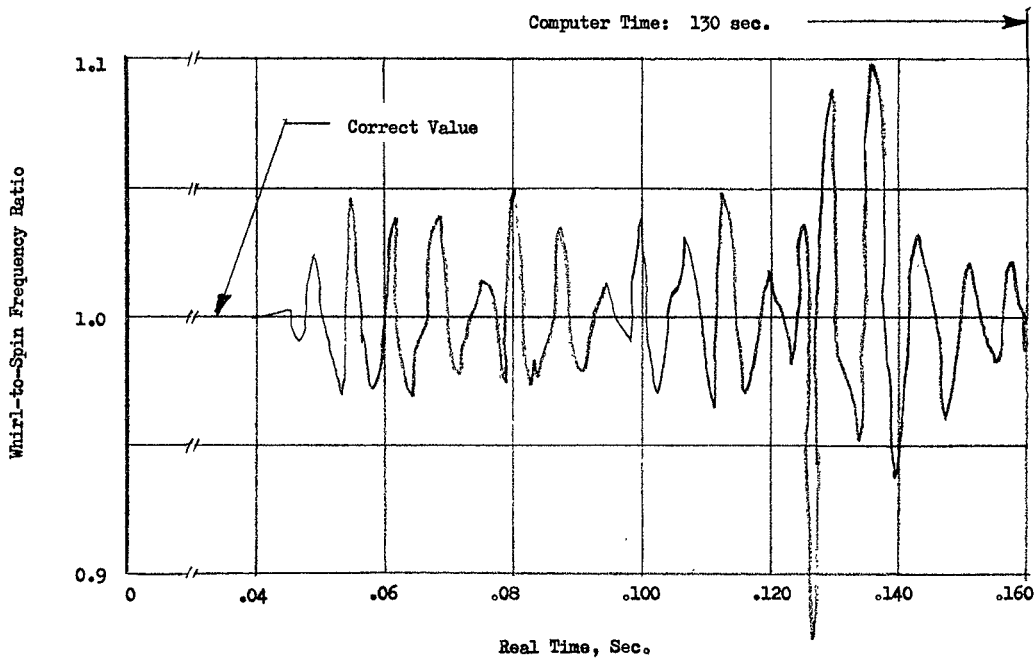


Figure 5 Data From a 5-Mass 95.5 rpm Flexible Rotor
in a Steady-State Operation

CAPABILITY AND LIMITATIONS OF THE CURRENT COMPUTER PROGRAM

Capability of the Computer Program

The Computer Program was written to include effects of:

1. Rotor elasticity in shear and bending
2. A multiplicity of non-linear stiffness bearings each in turn supported on a non-isotropic stiffness and damping characteristic mount
3. General in-phase and out-of-phase stiffness and damping functions
4. Mass and mass moments of inertia unbalance vectors
5. Drive torque and dissipation functions

The application of the program may be illustrated as follows:

1. Transient spin speed analysis through critical speed range using speed sensitive drive torque and/or dissipation functions
2. Various whirl-to-spin frequency mode simulations in transient, steady-state or quasi (cyclic) steady-state operations
3. Study of the effects of rotor transverse gravity or constant acceleration
4. Investigation of the effects of general fluid dynamic or some electromagnetic excitation function with in-phase or out-of-phase, whirl and/or spin velocity sensitive, stiffness and/or damping functions.
5. Non-linear bearing stiffness effects
6. Study the effects of shear-and/or bending-wise flexible couplings on rotordynamic performance
7. Analysis of a general rotor motion in a non-axisymmetric and non-steady state operation.

Limitations

Several major assumptions in the mathematical formulation of the simplified computer program were made. The assumptions and resulting limitations are itemized as follows:

Torsional Rigid Rotor Assumed. For most rotor design properties this assumption appears to be realistic. Deviations from the real solution would occur, only when substantial torsional oscillations coupled with transverse rotor motion are encountered. A multiple-rotor system with small dissipation coefficient and having pulsating drive torque of a frequency at, or near, a torsional critical speed of the rotor would be the case that a torsionally flexible rotor model would provide more accurate results.

Rotor Casing Parameter not Considered. For rotors with large casing-to-rotor mass ratio or reasonably rigid attachment between the casing and the foundation, the effects of the assumption are minimal.

Rotor Hysteretic Damping Including the Effects of Bolted or Press-fit Rotor Joints not Considered. This assumption does not materially affect the computation results unless an elastically bent rotor of substantial hysteretic property operates under a non-synchronous whirl condition, or under a substantial gravity or transverse acceleration loading.

COMPUTER PROGRAM USER'S INSTRUCTIONS

DESCRIPTION OF THE COMPUTER PROGRAM

The Simplified, Flexible Rotor, Transient-Speed Rotordynamics Analysis Computer program consists of a MAIN program, 12 subroutines and a major library subroutine (ISIMDD). The program is written for IEM 360, Model 65 and Fortran H compiler language. The CRT plotting language is to be compatible with that available at NASA Lewis Research Center, Cleveland, Ohio*. A brief description of the functions of the MAIN program and subroutines and a general flow diagram, Figure 6, is provided, as a guide for the use of the program.

*CRT Plotting Reference, Robert G. Kannenberg, Lewis Research Center, Cleveland, Ohio: "Cinematic-Fortran Subprograms for Automatic Computer Microfilm Plotting, NASA TM X-1866" November 1969.

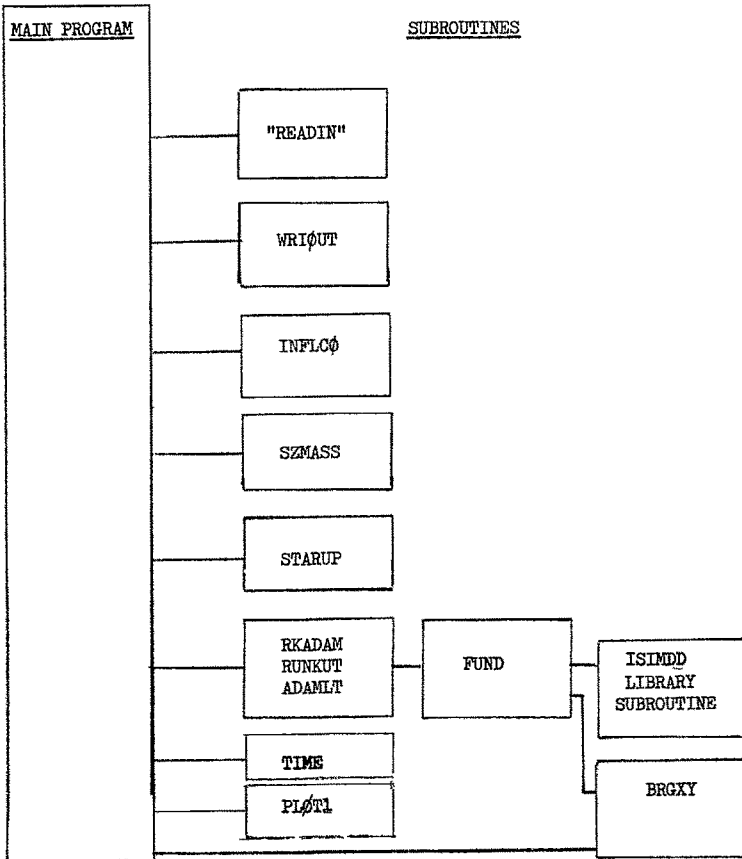


Figure 6 General Flow Diagram Relating Various Subroutines and the MAIN Program

PROGRAM OR SUBROUTINE NAME

FUNCTION

MAIN

This is the basic controlling program which originates the basic "call" statement. It also provides output write-out and CRT generating procedures. The rotor motion in stationary coordinates is converted into rotating coordinates before transmission to the integration subroutines. Similarly, the results of integration are first converted back to stationary coordinates before they are used in print-outs and CRT graphs.

READIN

This function is basically to *read input other than providing built-in average data values. "READIN" is called by "MAIN."

1. 2-card job descriptive title
2. NAMELIST/DATA 1/, the data which must be input
3. NAMELIST/DATA 2/, optional, when the input data in the data 2 group are not provided, the built-in average data values will be used.
4. Read punched cards from previous analysis for continued study. The read-in of the punch cards becomes effective when $C\phi NTIN = 1$ is specified in data as opposed to $C\phi NTIN = 0$ condition.

* Detail READIN information will be discussed in subsequent section.

PROGRAM OR SUBROUTINE NAME

FUNCTIONS

WRI@UT

This is to print out all the input data in groups according to their uses. Descriptive definitions and dimensional units are provided for each data set. "WRI@UT" is called by "MAIN."

STARUP

This subroutine is to generate accurate initial rotor motion conditions for starting computations in the basic program. Using the "STARUP" subroutine, computations in the basic program can proceed in an efficient manner. The present "STARUP" is in its simplified version; it includes only linear bearing and rotor excitation stiffness functions. Torque, damping and out-of-phase force parameters are not included. It is in a tentative status subject to updating to include all leftover parameters. "STARUP" is a steady-state subroutine and is called by "MAIN."

FUND

This subroutine provides Time derivatives required by the integration subroutines "RKADAM," "RUNKUT," and ADAMLT."

To compute the derivatives, "FUND" obtains input data through "READIN," influence coefficient data from "INFLC@," and bearing data from "BRGXY."

PROGRAM OR SUBROUTINE NAME

FUNCTIONS

RKADAM,
RUNKUT, &
ADAMLT

These integration subroutines are based on an adjustable step, predictor-corrector, Adam-Moulton integration procedure. The starting points are established through forward difference Runge-Kutta approach although option in backward difference procedure is also provided. "TOLI" is an input data name which is used to regulate the computation tolerance in integration accuracy as may be desired. Higher accuracy may be obtained with smaller "TOLI" value such as .001 or .0001. Also, higher accuracy means longer computation time. Thus, an optimum balance between the degree of accuracy and computation time should be made to suit the circumstances.

"RKADAM," "RUNKUT" and "ADAMLT" are called by "MAIN."

INFLCØ

This generates the required influence coefficients considering both bending and shear elasticity of the rotor. It is called by "MAIN" and the influence coefficients are also used by "STARUP" and "FUND."

ISIMDD

This is a simultaneous linear equation solution library subroutine which computes individual derivatives required by the integration subroutines. It is called by "FUND."

PROGRAM OR SUBROUTINE NAME

FUNCTIONS

SZMASS

This a subroutine which computes the rotor mass, transverse and polar moments of inertia, and rotor length coordinates. It is called by "MAIN."

BRGY

This is a subroutine which computes journal displacements from their respective zero-load position

The computations appropriately consider the nonlinear stiffness characteristics of the bearings, and non-isotropic mount force characteristics. "BRGY" is called by "MAIN" AND "FUND."

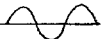
TIME

This is to furnish real time for the iteration process from the tentative "real time" output of the integration subroutines.

~~PL~~T1

It is to provide multiple bearing data for CRT plotting. It is used to generate journal deflections and bearing forces for a maximum number of 12 bearings. It is called by "MAIN."

INPUT PROCEDURE

In using the computer program a continuous rotor configuration is to be simulated by a discrete-mass rotor with appropriate massless elastic members in between adjacent rotor masses. In general, the minimum number of discrete masses used should be such that the rotordynamic mode shape can be possibly sustained. For instance, if a shaft operates in its third critical speed range the mode shape for a low stiffness bearing would be ; to sustain this mode a minimum of 5 masses is required. To obtain good accuracy in general, several times the minimum number of mass requirements are used. In rotor motion predominately influenced by mass eccentricity, damping, and stiffness function, the mode shape in a critical speed range may be substantially modified from that of a pure critical speed mode shape. Judgement must hence be exercised in selecting the number of discrete masses in representing a rotor configuration.

The rotor to be studied is first divided into consecutively numbered stations. The total number of stations may vary from 5 to 25, inclusively. Rotor sections between adjacent rotor stations are labeled with the same numbers as that of the left adjacent stations. The rotor property input data are appropriately subscripted according to the rotor station or section numbers. For non-linear stiffness bearing data two-dimensional subscripts are used. The first subscripts define their rotor station location and the second define the non-linear bearing stiffness sections.

The input data is printed prior to the writing of computational results

A detailed description of the input, output and usage of the program appears

in the following sections. To supplement the input procedure description, a sample input code sheet is shown in Figure 7.

Input

A namelist input procedure is adopted as the major input format due to its flexibility in selecting input parameters and the liberal use of built-in average values when desired. For preliminary analysis, by making use of built-in data, the input data volume can be drastically reduced.

The complete input listing is as follows:

1. Read Title. Two consecutively located 80-character spaced cards must be used for job description title. The title description may use the first 72 spaces of each card and 73 through 80th space may be used for card identification number only.
2. NAMELIST/DATA 1/. The data group consists of key input data which must all be read in Namelist format. This sequence of read-in data is immaterial. Because the Data 1 parameters pertain to the rotor geometry and other essential descriptive information, average-value data cannot be built into the program. The physical input of these parameters is necessary. The names and their definitions contained in the NAMELIST/DATA 1/ are on page 44.
3. Name List/Data 2/. The majority of the input data is included in the "Data 2" namelist. All parameters contained in "Data 2" are optional input data; i.e., they need not be inputted if the built-in average values are adequate. However, a card with the name Data 2 is necessary even if the data field is empty.

The input names for Data 2, their definitions, and the built-in values are on page 45.

4. Reading Punched Cards. If continuation of a previous analysis is desired specify "CONTIN=1" and attach the cards providing starting data from a previous computation.

CAUTION: If K is greater than 1 for one or more non-linear stiffness bearings, the input of double-subscripted $BR\phi(I,K)$ for appropriate K values must be input. Where I is the rotor station for the non-linear bearing, K varies from 1 through K for appropriate non-linear bearings. The input of $BR\phi(I,1)$ would be necessary when a different than built-in value of 0.005 inch is desired.

NAMELIST/DATA 1/

<u>Name Used In The Computer Program</u>	<u>Equivalent Name in The Mathe- matical Model</u>	<u>Stored Value</u>	<u>Description</u>
NS	N		Number of rotor stations (allowable range: $5 \leq NS \leq 25$).
TMAX			Total real time to be run, sec.
TSTØP			The computer time allowed for each set of data, minutes.
DD(NS-1)*			Outside diameters of rotor sections between adjacent rotor stations, in.
QL(NS-1)	l_i		Rotor section lengths between adjacent rotor stations, in:
NB			Number of non-linear stiffness bearings. (Allowable range: $2 \leq NB \leq 12$)
IB(NB)			Rotor station numbers of non-linear stiffness bearings
K(NB)			Total number of stiffness sections for each of the non-linear stiffness bearings. (Allowable range: $1 \leq K \leq 6$)
ICØND			ICØND=1 Means starting a new rotor-bearing configuration. ICØND=0 Means read-in alternate initial conditions for the same rotor-bearing configuration.
CØNTIN			CØNTIN=0 Means starting a new rotor-dynamics analysis with initial conditions provided by the startup subroutine CØNTIN=1 Means continuation of a previous analysis by using the previous results on punched cards as the initial conditions.
FDØT			Initial rotor spin frequency, rpm.
WHIVEL			Initial rotor whirl frequency, rpm.

* (NS), (I) or (I, J) after a name specifies the name being an array of one or two dimension. The values of the indexes specify the current size of the array.

NAMELIST/DATA 2/

Name Used In The Computer Program	Equivalent Name in The Mathe- matical Model	Stored Value	Description
T		0	Initial real time, sec.
DT		.00001	Estimated initial integration step (real) time, sec. The computer program may modify the value of DT to suit tolerance requirements.
FD		0	Rotor spin angular displacement co-ordinate, degrees, (This value is added to data).
IASIGN		1	Rotor station at which whirl/spin frequency ratio will be plotted on CRT.
N ϕ RPM		1	The number of spin speeds in rpm at or near which 3-dimensional absolute rotor mode shape CRT graphs are required. The spin speed rpm values are listed under INPRPM array. (Allowable range: $0 \leq N\phi RPM \leq 50$).
NP ϕ INT		25	The number of points (one per integration step) for each CRT graph. (Allowable range: $1 \leq NP\phi INT \leq 50$). It applies to all CRT graphs except the 3-dimensional rotor mode shape plot.
CRT		0	CRT=0 Means CRT is not required, CRT=1 Means CRT is required.
ACCEL		0	ACCEL=0 Means a 3-dimensional steady-state rotor mode shape CRT would be provided if concurrently CRT=1. ACCEL=1 Means the steady-state rotor mode shape will not be provided.
INPRPM(N ϕ RPM)		0	The rotor spin speed rpm values at or near which CRT graphs for 3-dimensional absolute rotor mode shapes are required.
T ϕ LI		.01	Integration tolerance, fraction.
T ϕ LB		.001	Tolerance in computing bearing displacements, fraction.
GX	g_x	0	Gravity or G-loading in X direction, in/sec ² .

NAMELIST/DATA 2/

<u>Name Used In The Computer Program</u>	<u>Equivalent Name in The Mathe- matical Mode</u>	<u>Stored Value</u>	<u>Description</u>
GY	g_y	0	Gravity ₂ or G-loading in Y direction, in/sec .
TMZ	M_z	0	Exponent for speed sensitive rotor drive torque
TMZ1	M_{z1}	0	Coefficient for rotor drive torque
TMZ2	M_{z2}	0	Coefficient for rotor drive torque, (in-lb-sec)/rad
TMZ3	M_{z3}	0	Coefficient for rotor drive torque, in/lb.
D(NS-1)		0	Inside diameters of rotor sections between adjacent rotor stations, in.
DN(NS-1)		.283	Material densities of rotor sections between adjacent stations, lb/in ³ .
EE(NS-1)		3×10^7	Young's moduli of rotor sections between adjacent stations, lb/in ² .
GG(NS-1)		1.15×10^7	Shear moduli of rotor sections between adjacent stations, lb/in ² .
EI(NS-1)		0	Direct input of the products of Young's moduli and area moments of inertia of rotor sections between adjacent stations, lb-in ² .
*GAK(NS-1)		0	Direct input of the products of shear moduli, cross-sectional areas and reciprocals of shear stress concentration factors between adjacent stations, lb.
AM(NS)		1×10^{-16}	Additional rotor masses at rotor stations, (lb-sec ²)/in.
AID(NS)		0	Additional rotor transverse mass moments of inertia at rotor stations, (lb-in-sec ²).
AIRØ(NS)		1×10^{-16}	Additional rotor polar mass moments of inertia at rotor stations, (lb-in-sec ²).

* GAK stands for $\frac{(GG)(\text{Rotor Cross-section Shear Area})}{K}$, where K denotes

Maximum to average shear stress ratio expressible as:
$$\frac{1}{3} \frac{[(DD)^3 - D^3]}{[(DD)^2 + D^2](DD-D)}$$

NAMELIST/DATA 2/

<u>Name Used</u> In the Computer Program	<u>Equivalent</u> Name in The Mathe- matical Model	<u>Stored</u> <u>Value</u>	<u>Description</u>
ECC(NS)	e_i	1×10^{-16}	Rotor mass eccentricities at rotor stations, in.
ALFA(NS)	α_i	0	Phase angles for rotor mass eccentricity vectors at rotor stations measured from that of the first rotor station, degrees.
BETA(NS)	β_i	0	Initial misalignments between the axes of the mass moments of inertia and the elastic axes at rotor stations, degrees.
GAMMA(NS)	γ_i	0	Angular positions of the X-Y plane projections of the axes of mass moments of inertia at rotor stations measured from that at the first rotor station, degrees.
CZ(NS)	C_{Zi}	0	Torsional friction exponents at rotor stations, dimensionless.
CZ1(NS)	C_{Z1i}	0	Torsional friction coefficients at rotor stations, dimension of $C_{Z1i} (FD/\phi T) C_{Zi}$ in-lb.
CZ2(NS)	C_{Z2i}	0	Torsional friction coefficients at rotor stations, (in-lb-sec)/rad.
XKF(NS)	$K_{\phi i}$	0	Whirl-frequency factors for stiffness force coefficients at rotor stations, dimensionless.
XCF(NS)	$C_{\phi i}$	0	Whirl-frequency factors for damping force coefficients at rotor stations, dimensionless.
XKFF(NS)	$K_{\phi Fi}$	0	Whirl-frequency factors for stiffness moment coefficients at rotor stations.
XCFF(NS)	$C_{\phi Fi}$	0	Whirl-frequency factors for damping moment coefficients at rotor stations, dimensionless.
QK(NS)	K_i	0	In-phase stiffness force coefficients at rotor stations, lb/in.
QC(NS)	C_i	0	In-phase damping force coefficients at rotor stations, (lb-sec)/in.

NAMELIST/DATA 2/

<u>Name Used</u> In The Computer Program	<u>Equivalent</u> Name in The Mathe- matical Model	<u>Stored</u> <u>Value</u>	<u>Description</u>
QKP(NS)	K_{Pi}	0	Out-of-phase stiffness force coefficients at rotor stations, lb/in.
QCP(NS)	C_{Pi}	0	Out-of-phase damping force coefficients at rotor stations, (lb-sec)/in.
QKHD(NS)	K_{HDi}	0	Out-of-phase whirl and spin velocity sensitive stiffness force coefficients at rotor stations, (lb-sec)/(in-rad).
QCHD(NS)	C_{HDi}	0	Out-of-phase whirl and spin velocity sensitive damping force coefficients at rotor stations, (lb-sec ²)/(in-rad).
QKF(NS)	K_{Fi}	0	In-phase stiffness moment coefficients at rotor stations, in-lb.
QCF(NS)	C_{Fi}	0	In-phase damping moment coefficients at rotor stations, in-lb-sec.
QKPF(NS)	$K_{\phi Pi}$	0	Out-of-phase stiffness moment coefficients at rotor stations, in-lb.
QCPF(NS)	$C_{\phi Pi}$	0	Out-of-phase damping moment coefficients at rotor stations, in-lb-sec.
QKHDF(NS)	$K_{\phi HDi}$	0	Out-of-phase whirl and spin velocity sensitive stiffness moment coefficients at rotor stations, (lb-in-sec)/rad.
QCHDF(NS)	$C_{\phi HDi}$	0	Out-of-phase whirl and spin velocity sensitive damping moment coefficients at rotor stations, (lb-in-sec ²)/rad.
BKMX(NB)	K_{mxi}	10^{10}	Non-isotropic mount stiffness coefficients in X and Y-direction, respectively, for non-linear stiffness bearings, lb/in.
BKMY(NB)	K_{myi}		
BCMX(NB)	C_{mxi}	1×10^{-16}	Non-isotropic mount damping coefficients in X-direction for non-linear stiffness bearings, (lb-sec)/in.
BCMY(NB)	C_{myi}	1×10^{-16}	Non-isotropic mount damping coefficients in Y-direction for non-linear stiffness bearings, (lb-sec)/in.
BCB(NB)	C_{Bi}	1×10^{-16}	Bearing damping coefficients for non-linear stiffness bearings, (lb-sec)/in.

<u>Name Used In The Computer Program</u>	<u>Equivalent Name in The Mathe- matical Model</u>	<u>Stored Value</u>	<u>Description</u>
*FDØFIX	ϕ_o	0	A bearing stiffness speed parameter, rpm
*BNB(NBK)	B_{NLK}	0	Rotor spin-speed sensitive bearing stiffness coefficients for K stiffness sections of non-linear stiffness bearing.
*BBB(NB, K)	B_{BiK}	0	Rotor spin-speed sensitive bearing stiffness coefficients for K stiffness sections of bearings for non-linear stiffness bearing.
*BRØB(NB, K)		.005 (for the 1st stiffness sec- tions of all bearings)	Upper bearing-displacement limits for K stiffness sections of bearing, in.
*BKB(NB, K)	K_{BiK}	1×10^6	Non-linear bearing stiffness coefficients for K stiffness sections of bearing.
*BHB(NB, K)	H_{BiK}	1	Non-linear bearing stiffness exponents for K stiffness sections of bearing.
*BDB(NB, K)	D_{BiK}	0	Non-linear bearing stiffness coefficients for K stiffness sections of bearing.
*BEB(NB, K)	E_{BiK}	0	Non-linear bearing stiffness coefficients for K stiffness sections of bearing.

*See equations on p. 60

FORTRAN FIXED 10 DIGIT DECIMAL DATA

DECK NO. _____ PROGRAMMER _____ DATE _____

PAGE _____ OF _____ JOB NO. _____

NUMBER	DESCRIPTION	
1	A C H E C K O U T R	
13	T O R C O N F I G U	
25	R A T I O N C O N S I	
37	S T S O F 5 S T A	
49	T I O N S A N D 5	
61	M A S S E S R O T O	
IDENTIFICATION 73		80

1	R M A S S U N B A L	
13	A N C E S I N C L U D	
25	E D . B E A R I N G S	
37	A R E A T S T A T	
49	I O N S 1 & 5 . A	
61	P R I L / 2 7 / 1 9 7 0	
IDENTIFICATION 73		80

1	& D A T A 1 N S = 5	
13	, T M A X = 1 , T S T O	
25	P = 1 , D D = 4 * 1 , Q	
37	L = 4 * 1 , N B = 2 , I	
49	B = 1 , 5 , I C O N D =	
61	1 , C O N T I N = 0 , F	
IDENTIFICATION 73		D O T = 1 0 0 , 80

1	W H I V E L = 1 0 0 ,	
13	K (1) = 1 , K (5) =	
25	1 , & E N D	
37		
49		
61		
IDENTIFICATION 73		80

NUMBER	DESCRIPTION	
1	& D A T A 2 C R T =	
13	1 , A C C E L = 1 , I N	
25	P R P M = 2 , & E N D	
37		
49		
61		
IDENTIFICATION 73		80

1	/ *	
13		
25		
37		
49		
61		
IDENTIFICATION 73		80

1		
13		
25		
37		
49		
61		
IDENTIFICATION 73		80

1		
13		
25		
37		
49		
61		
IDENTIFICATION 73		80

50

Special Notes on Input

1. Alternate Provisions. In the input procedure there are two alternate input provisions. In certain rotor designs the flexibility of rotor may not be derived directly from the diameter and elastic modulus considerations, such as complex bolted rotor sections. In such circumstances, an optional combined EI or GAK may be more conveniently used in place of EE, DD, D or GG and shear stress concentration factor.

EI is the product of Young's modulus of elasticity and sectional moment of inertia, GAK is the product of shear elastic modulus, rotor cross-sectional area and reciprocal of maximum to average shear stress ratios, where

EE = Young's modulus of elasticity, psi
GG = Shear modulus of elasticity, psi
DD = Outside rotor diameter, in
D = Inside rotor diameter, in

In using the option input, EI, EE set or GAK, GG set, only values of one of the two sets can be assigned. No values are to be assigned to the other set.

The built-in average values of EE and GG for all rotor sections are:

$$\begin{aligned} EE (I) &= 3 \times 10^7 \\ GG (I) &= 1.5 \times 10^7 \end{aligned}$$

It should be noted that the option, EI or GAK, may be used for any rotor sections desired, and need not be used for all rotor sections.

2. Analysis with Various Alternate Operating Parameters. The program has the provision to analyze the effect of various sets of operating parameters on the performance of a given rotor-bearing

configuration. This can be accomplished by inputting; first, a complete data set including the rotor design and the initial set of values of operating parameters including "ICOND=1" in Data 1/Name List; the second set of data for different values of certain parameters, but not including "ICOND=1", may be input without repeating the same rotor design information. This procedure may be continued indefinitely until the analysis of a new rotor design is initiated. With the new rotor design data, "ICOND=1" must be included in the first data input set.

The rotor-bearing configuration, referred to above, is defined as:

- a. Rotor design, geometry and related physical properties which include the number of rotor stations (NS), transverse rotor gravity or accelerations (GX, GY), rotor diameters, length, density, elasticity, mass, and mass moments of inertia.
- b. Bearing locations and number of bearings used.

The parameters other than that of rotor-bearing configurations may be repeated with different numerical values for a given rotor-bearing configuration. They are listed as:

- a. Initial rotor motion conditions.
- b. Bearing and mount damping and stiffness characteristics, number of non-linear stiffness sections.
- c. Unbalance parameters for rotor masses and mass moments of inertia.
- d. Rotor drive torque and torsional damping parameters.
- e. All rotor in-phase and out-of-phase stiffness and damping parameters.

Namelist Input Procedure and Sample Input Data Sheet

According to the current North American Rockwell Corporation practice compatible with IBM 360 Fortran IV, compiler H language, the namelist input format is described in the following five pages.

A sample of a completed input data sheet is given in Appendix A.

NAMelist INPUT-OUTPUT

Namelist offers a quick and relatively easy way of coding input-output. The programmer writes a `NAMelist` statement, including one or more lists of variables and/or arrays he wishes to read in or print out, and assigns a name to each list. Thereafter, instead of rewriting an entire list, he references the list name in `READ` and `WRITE` statements.

On input, the number of items to be read and the place each one is to be stored is governed to a large extent by the input data cards. Each item is written in a format much like an arithmetic statement, with each variable explicitly named and set equal to the desired value. A few items or many can be read by the same `READ` statement, and the order of writing the data is unimportant.

On output, the members of the list will be printed in a standard format. The appearance of the printout is not suitable for formal reports, but is satisfactory as a substitute for a partial dump in error branches, for printout of input data, for intermediate results during checkout, or in any other case in which answers are needed and the format is not important.

TRANSMITTAL STATEMENTS

General Form:

```
READ (a, b, END = c, ERR = d)
```

```
WRITE (a, b)
```

Where: a is the data set reference number.

b is the Namelist name.

See Section 204.2, Sequential Input-Output, for a complete description of these statements.

Note that Namelist cannot be used with a `PUNCH` statement. However, the statement

```
WRITE (14, b)
```

can be used to produce punched output.

NAMelist STATEMENT

General Form:

```
NAMelist /x/ a, b, ..., c /y/ d, e, ..., f  
/z/ g, h, ..., i
```

Where: x, y, and z are Namelist names.

a, b, c, ... are variable or array names.

The following rules apply to construction of a `NAMelist` statement.

1. A Namelist name consists of from 1 through 6 alphanumeric characters, the first of which is alphabetic. The name is enclosed in slashes.
2. A Namelist name may be defined only once by its appearance in a `NAMelist` statement, and it must be so defined before its use. After it is defined in the `NAMelist` statement, the Namelist name may appear only in input or output statements thereafter in the program.
3. A `NAMelist` name cannot be transmitted as an argument from a calling program to a subprogram. In a subprogram, dummy arguments cannot appear in a Namelist list.
4. The list of variable and array names belonging to a Namelist name ends with a new Namelist name (enclosed in slashes) or with the end of the `NAMelist` statement.
5. A variable or array name may belong to one or more Namelist names. For example:

```
DIMENSION A(10), I(5), L(6)  
NAMelist /NAM1/ A, D, I, K, L  
/NAM2/ A, C, K, M
```

means that the arrays A, I, and L, and the variables D and K are included in the list for `NAM1`, and the array A and the variables C, K, and M are associated with the Namelist name, `NAM2`.

6. The specification statement(s) defining the type, size, and relative storage locations of arrays and variables must precede any `NAMelist` statement in which those arrays and variables are mentioned.

INPUT DATA

The input data must be in a special form in order to be read with a Namelist read statement. Following are the published IBM specifications for the data:

1. The first character of each input record (card image) must be blank.
2. The second character in the first record (of a group of records) must be an &.
3. The Namelist name must start in column 3 of the first record of the group.
4. The Namelist name must be followed by a blank; it must not contain embedded blanks.
5. Input data items follow the Namelist name and are separated by commas. There must not be embedded blanks in the variable names, array names, or constants.
6. Input data items can be continued on succeeding records. A comma after the last item of data on a record is optional. Each succeeding record must begin with a complete variable name, array name, or constant (not with an equals sign).
7. The end of a group of data items is signaled by &END, anywhere in a record except in the first character position.
8. Constants in the data items may take any of the following forms:
 - a. Integer
 - b. Real
 - c. Complex
 - d. Logical
 - e. Literal (input only)

Constants are written exactly as in a source program, except that logical constants may be written in the form T or F as well as .TRUE. or .FALSE..

9. The form of the data items may be:

Variable name = single constant

The variable name must be one of the names in the Namelist list. It may be a single variable name or it may be subscripted.

Array name = set of constants

The array name must be one of the names in the Namelist list.

The set of constants may be separated by commas, or may be in the form "k*constant" where k is an unsigned integer used to represent k constants.

If the array name is not subscripted, the first of the set of constants will be stored in the first element of the array, with subsequent constants stored in consecutive elements.

The number of constants must be less than or equal to the number of elements in the array.

10. If an item that appears in a Namelist list also appears in an EQUIVALENCE statement, only the name from the list may appear on an input record.

The following items are comments on the current IBM Namelist implementation (Release 15/16). As such, they are subject to change in future releases.

1. The first character of any input record is ignored.
2. Embedded and trailing blanks in a constant or exponent are treated as zeros. For example:

Input Data Item

I = 4o,
J = 4o3,
E = 1.0E6o,
K = ,

Is Interpreted As

I = 40,
J = 403,
E = 1.0E60,
K = 0,

3. Hexadecimal constants, of the form Znnnnnnnn, are accepted. (On output, they will be assigned the type of the name in the list.)
4. Literal constants, of the form 'aaaa', are accepted. (On output, they will be assigned the type of the name in the list.)

When using Namelist to read a literal constant into an array, all items in the array following the literal constant will be set to EBCDIC blanks. Thus, if there are three words in the array ARR, the input data item

```
ARR(1) = 'ABCD'
```

will store the characters ABCD in the first word and will fill the next two words with blanks. The input data item

```
ARR(1) = 'ABCDEFGHIJKL'
```

will fill all three words in the array with the characters shown.

5. The following form is permitted:

Subscripted array name = set of constants

When the array name is subscripted, the first item of the set of constants will be placed in the element indicated, with subsequent constants in consecutive elements.

6. The form

Array name = single constant

is interpreted to mean that the single constant is to be put in the first element of the array.

7. A repetition factor can be inserted anywhere in the list. Assume that the array TERSE has a dimension of 10. The input data items to fill this array could be:

```
TERSE = 4.0, 8*0.0, 6.0
```

The first and last elements of the array will be set to 4.0 and 6.0, respectively; the intervening elements will be set to 0.0.

8. If the size of an array is exceeded, the error messages appear irrelevant. Depending on the context, the following might be printed:

NAME NOT IN NAMELIST DICTIONARY
or
NAME LARGER THAN 8 CHARACTERS

9. If a real number is set equal to an integer, the decimal point is assumed to be at the end of the number.

<u>Input Data Item</u>	<u>Is Interpreted As</u>
REAL = 1234,	REAL = 1234.0,

10. If an integer is set equal to a real number, the decimal point is ignored.

<u>Input Data Item</u>	<u>Is Interpreted As</u>
INTGR = 2.345,	INTGR = 2345,

11. If an item is repeated in the group of Namelist input records, the last one read is used.

12. Since the last item on a card must be a data name or a constant, an identification field may not be punched in columns 73-80.

13. If EEH facilities (Region 226) are available, a CALL ERRSET statement will permit continuation of the program after Namelist errors are encountered. For example, the statement

```
CALL ERRSET (221, 20, 20, 0, 1, 224)
```

will permit up to 19 occurrences each of errors 221 through 224; the program will be terminated on the 20th error for any one of these error numbers. A message showing the card in error will be printed.

Note that the standard fixup ignores all remaining items in the group of input records. Not only the erroneous item, but all the following items in the group are ignored.

INPUT PROCEDURE

When a READ statement references a Namelist name, the input of data is begun. The first data card (or record) is read and examined to verify that its name is consistent with the Namelist name in the READ statement.

If the specified Namelist name is not found, additional records are examined consecutively until there is a successful match, or until all the data for the program are exhausted. (This provides a method for skipping undesired blocks of data in some branches of the program.)

Reading continues until an &END is encountered. Any information following the &END is ignored.

OUTPUT

When the arrangement of the printed page is not significant, Namelist relieves the programmer of the effort of setting up formats.

When a WRITE statement references a Namelist name:

1. All variables and arrays, and their values, belonging to the Namelist name will be written, each according to its type. Arrays are written in columnwise order.
2. The output data will be written such that:
 - a. The field for the data will be large enough to contain all significant digits.
 - b. The output can be read by an input statement referencing the Namelist name.

The PUNCH statement cannot be used with Namelist. However, a WRITE (14, nname) statement, where nname is a Namelist name, can be employed to produce punched output. A suitable DD statement is also required; one is provided in AFSLINK.

EXAMPLE

Assume that the program contains the following statements:

```
COMPLEX COMPLX (5)
REAL*8 DBLE (5)
REAL*4 REAL (10)
INTEGER INTGR (10)
LOGICAL LOGCL (20)
```

```
NAMelist /INPUT/ REAL, INTGR,
          COMPLX, DBLE, LOGCL
```

```
READ (5, INPUT)
```

Also assume the four data cards in the right column. (For the example, the input is written in the same order as the Namelist list. This is for convenience comparing the input to the output; the data need not be written in that order.)

The first record is read from the device associated with data set reference number 5 (normally the input stream). The record is searched for an & in column 2, immediately followed by the Namelist name, INPUT, and the required blank after the name.

Since the search is successful, the data items are converted and placed in core. If desired, only one or two of the variable names could have appeared in the input records.

The ten real constants will be placed in REAL(1) through REAL(10). The constant 2 will be placed in INTGR(1), the constant 6 will be stored in INTGR(2) through INTGR(9), and the constant 10 will be stored in INTGR(10).

The arrays COMPLX and DBLE will be similarly filled as indicated. The first two items of the array LOGCL will be set to .TRUE.; the remainder of that array will be set to .FALSE..

INTGR(3), previously set to 6, will be reset to 9 by the last data item.

The &END signals termination of the input for this READ statement.

The example on the following page shows output produced by the statement:

```
WRITE (6, INPUT)
```

For the illustration, the output data are derived from the input shown below.

1	&INPUT	
13	REAL=1.1414,	
25	2.704,3.945,	
37	85.31,99.96,	
49	95.3,333.E12	73
61	,1886E9,	
1	169017.E43,	
13	6627506,	
25	INTGR=2,	
37	8*6,10,	
49	COMPLX=(1.0,	73
61	2.0)	
1	4*(2.2986,	
13	4.77),	
25	DBLE=1.04,	
37	1.5578552,	
49	1.41427854,	73
61	.13534D06,	
1	.1484167D9,	
13	LOGCL=T,.TRU	
25	E.,18*F,	
37	INTGR(3)=9,	
49		73
61	&END	

EXAMPLE OF NAMELIST OUTPUT

```

&INPUT
REAL= 1.J413994      , 2.7C39995      , 3.9449997      , 85.309998      , 99.959991      ,
95.29797d      , 0.33279992E 15, 0.18859992E 13, 0.16971693E 49, 6627506.0      ,
INTGR=      5,      6,      6,      9,      6,      6,
      (1.CCCCCC,2.CCCCCC),      (2.2985992,4.7699995),
      (2.2915992,4.7699995),      (2.2985992,4.7699995),
      (2.2985992,4.7699995),DBLE= 1.C4CCCCCCC0000      ,
1.55785570CCCC30      , 1.41427854CCCC000      , 135340.C00000000      ,
14841670.CC3CCCO      ,LOGCL=T,T,F,F,F,F,F,F,F,F,F,F,F,F,F,F,F,F,F,F,F,F
&END
```

Standard Namelist printing was modified by a job control statement to reduce the width of the output to fit on this page. The JCL statement used was:

```
//G.FT06F001 DD SYSOUT=A,DCB=(RECFM=VBA,LRECL=90,BLKSIZE=990)
```

If this technique is employed to reduce the width of the output, all records written on the data set defined by data set reference number 6 must fit within the reduced-record length.

The maximum CTC block size for an ASP system is 1012.

Non-Linear Bearing Force Characteristic Representation

Provision is made in the program to simulate non-linear bearing stiffness characteristics; however, linear damping is used in the program. The non-linear bearing stiffness characteristics including simultaneously linear speed sensitive stiffness parameters may be represented with a mathematical formulation as follows: (To eliminate translation of parametric notations from that used in the mathematical formulation to that used in the computer program, the latter notations are used here).

$$\left[(FD\cancel{\phi}T - FD\cancel{\phi}FIX) \left(ENB + BBB (\rho - ER\cancel{\phi}B) \right) + EKB \right] (\rho^{HBB} + (BDB)\rho + BEB) = F$$

where ρ is the resultant journal displacement from its zero-load position and F is the corresponding resultant bearing force.

ENB used in the computer program was denoted in the mathematical formulation portion of this report by N_{Bik} , and similarly

BBB by B_{Bik}

ER $\cancel{\phi}$ B by ρ_{Bik}

EKB by K'_{Bik}

HBB by H_{Bik}

BDB by D_{Bik}

BEB by E_{Bik}

FD $\cancel{\phi}$ T by $\dot{\phi}$

FD $\cancel{\phi}$ FIX by $\dot{\phi}_0$

"FD $\cancel{\phi}$ FIX" is a non-subscripted name in the current program. It could be extended to a subscripted array in the future extension of the computer program. To simplify the expression of the above equation, the subscripts

i, k are omitted. The constants FDOFIX, ENB, BEB, etc., may be considered as that for a particular non-linear stiffness section of a bearing station. To briefly describe the method of determining the constants an arbitrary non-linear bearing stiffness characteristic is used as shown in Fig. 8.

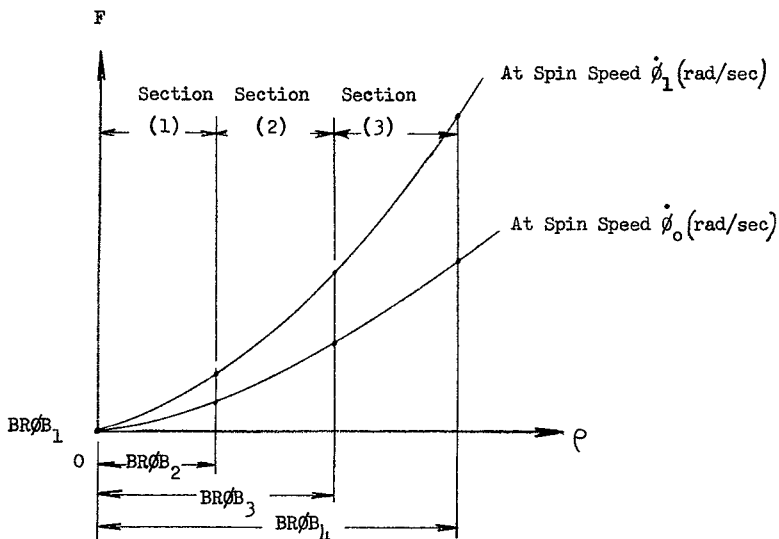


Fig. 8. Bearing Stiffness Curves

Assume the non-linear bearing stiffness section (2) is to be simulated. First use the non-linear stiffness characteristic at spin speed $\dot{\phi}_0$. Determine BHB according to the curvature, then solve for the three unknowns from the three equations,

$$BKB \left[\rho_i^{BHB} + (BDB) \rho_i + BEB \right] = F_i \quad i = 1, 2, 3$$

corresponding to three selected points on the characteristic curve.

The reason for using BKB as a common factor is that BKB may be directly used as a linear stiffness factor for a linear bearing when BHB = 1, and BDB = BEB = 0. Having determined the values of BKB, BHB, BDB and BEB, the values of BNB and BBB can be readily computed from selected two ρ values on the upper curve at spin speed $\dot{\phi}_1$ in Fig. 8. The equations used to obtain the solutions are:

$$\left\{ (\dot{\phi}_1 - \dot{\phi}_0) \left[\text{BNB} + \text{BBB}(\rho_1 - \text{BR}\phi_{B_2}) \right] + \text{BKB} \right\} \left[\rho_1^{\text{BHB}} + (\text{BDB}) \rho_1 + \text{BEB} \right] = F_1$$

$$\left\{ (\dot{\phi}_1 - \dot{\phi}_0) \left[\text{BNB} + \text{BBB}(\rho_2 - \text{BR}\phi_{B_2}) \right] + \text{BKB} \right\} \left[\rho_2^{\text{BHB}} + (\text{BDB}) \rho_2 + \text{BEB} \right] = F_2$$

In a similar manner the non-linear bearing stiffness representation of all other sections may be made.

Output Format

To provide a permanent record, the input data are printed out prior to the computation results.

The results of computations are printed with CRT graphs as optional additional output.

1. Print-Out. The computation results are printed out in the following sequence:
 - a. Total rotor weight, lb
 - b. Total rotor mass, the sum of additional rotor mass (AM) and the rotor mass computed from the rotor geometry (lb - sec²)/in
 - c. Total rotor polar mass moment of inertia, the sum of additional rotor polar mass moment of inertia (AIR ϕ) and the polar mass moment of inertia computed from the rotor geometry.
 - d. Force and moment influence coefficients
 - e. Initial rotor motion conditions
 - f. Elements in rotating coordinates consisting of:
 - 1) NS number of rotor displacement vector length, in
 - 2) NS number of rotor whirl displacements, rad
 - 3) NS number of rotor displacement vector velocities, in/sec
 - 4) NS number of rotor whirl velocities, rad/sec
 - 5) Rotor spin displacement rad
 - 6) Rotor spin velocity rad/sec
 - g. The appropriate derivatives consisting of:
 - 1) NS number of rotor displacement vector velocities in/sec
 - 2) NS number of rotor whirl velocities rad/sec
 - 3) NS number of rotor displacement vector radial accelerations in/sec²

- | | | |
|----|--|----------------------|
| 4) | NS number of rotor whirl accelerations | rad/sec ² |
| 5) | Rotor spin velocity | rad/sec |
| 6) | Rotor spin acceleration | rad/sec ² |
- h. X displacement array in stationary coordinates, (in.)
- i. Y displacement array in stationary coordinates, (in.)
- j. Rotor deflection vector array, in.
- k. Phase angle array for rotor deflection vectors from x-axis in the direction of rotation, degrees
- l. X velocity array in stationary coordinates, in.
- m. Y velocity array in stationary coordinates, in.
- n. Whirl frequency array, rpm
- o. Whirl-to-spin frequency ratio array
- p. Total number of revolutions
- q. Spin speed, rpm
- r. X bearing displacement coordinate, in.
- s. Y bearing displacement coordinate, in.
- t. X bearing velocity coordinate, in/sec.
- u. Y bearing velocity coordinate, in/sec.
- v. Journal displacement vector from bearing-center array, in.
- w. Journal displacement vector phase angle in the direction of rotation from X-axis, degrees
- x. Bearing reaction array, lb.
- y. Bearing reaction vector phase angle in the direction of rotation from x-axis, degrees
- z. Bearing force to journal deflection ratio, or equivalent linear bearing stiffness, lb/in/bearing.

A sample of printout of the computation results including that of input data is attached in Appendix A.

2. GRT Plotting Output. To facilitate the direct comprehension of the computation results and appreciation of the trend of rotordynamic

behavior, a series of 7 types of CRT are provided as an optional output controlled by an input word, "CRT." That is, CRT=1 for CRT output and CRT=0 for no CRT output.

The 7 types of CRT graphs are described as follows:

- a. Rotor 3-dimensional mode shape at or near preselected input speeds, "INPRPM," for a rotor operating with spin-speed acceleration or deceleration. For a steady-state rotor spin speed, one CRT of this type is provided during the run. The spin speed, at which the mode shape is depicted, is labeled on top of the graph. Three-dimensional information of the graph is provided by labeling the phase angle at corresponding displacement vector stations.
- b. Rotor spin speed versus time graph.
- c. Rotor whirl-to-spin speed ratio versus spin speed graph for a pre selected rotor station as designated by the input, "IASIGN."
- d. Bearing forces versus rotor spin speed graphs for all non-linear stiffness bearings.
- e. Bearing displacements versus spin speed graphs for all non-linear stiffness bearings.
- f. Maximum rotor displacement versus spin speed. The rotor station number for the maximum rotor deflection station for a spin speed is appropriately labeled for identification
- g. Rotor displacement at station "IASIGN" versus spin speed.

The number of points included in a frame of the CRT graph may vary from 1 through 50 as specified by the value of the input word, "NP/INT."

EXPERIMENTAL PROGRAM

The purpose of Task II, Mark 25 Rotor Dynamics Tests, was to perform spin tests utilizing the Mark 25 pump rotor to obtain data for correlation with the results of the rotordynamics computer program.

The test program included an initial series of seven tests to minimize rotor unbalance, plus eight investigative tests on the Mark 25 pump rotor with an array of unbalance levels and locations. Instrumentation was provided to measure response amplitudes and phase angles.

MARK 25 TURBOPUMP

The Mark 25 pump is an axial-flow liquid hydrogen pump consisting of an inducer and four axial stages. The pump rotor is mounted by duplex paired bearings. The bearings used are 55-millimeter bore size. To maintain radial stiffness required for rotordynamic considerations (critical speed, rotor whirl, and minimum tip clearance without rubbing), the bearings are arranged in axially preloaded pairs, each pair free to move independently of the other.

The rotor is built-up rotor of eight disks clamped together with 18, (5/16) through bolts. The overhung inducer is mounted by 6, (3/8) bolts to the forward disk stub shaft. The rotor is driven through a ball spline coupling at the aft stub shaft. The overall rotor length is approximately 24 inches with a nominal diameter of 6.1 inches.

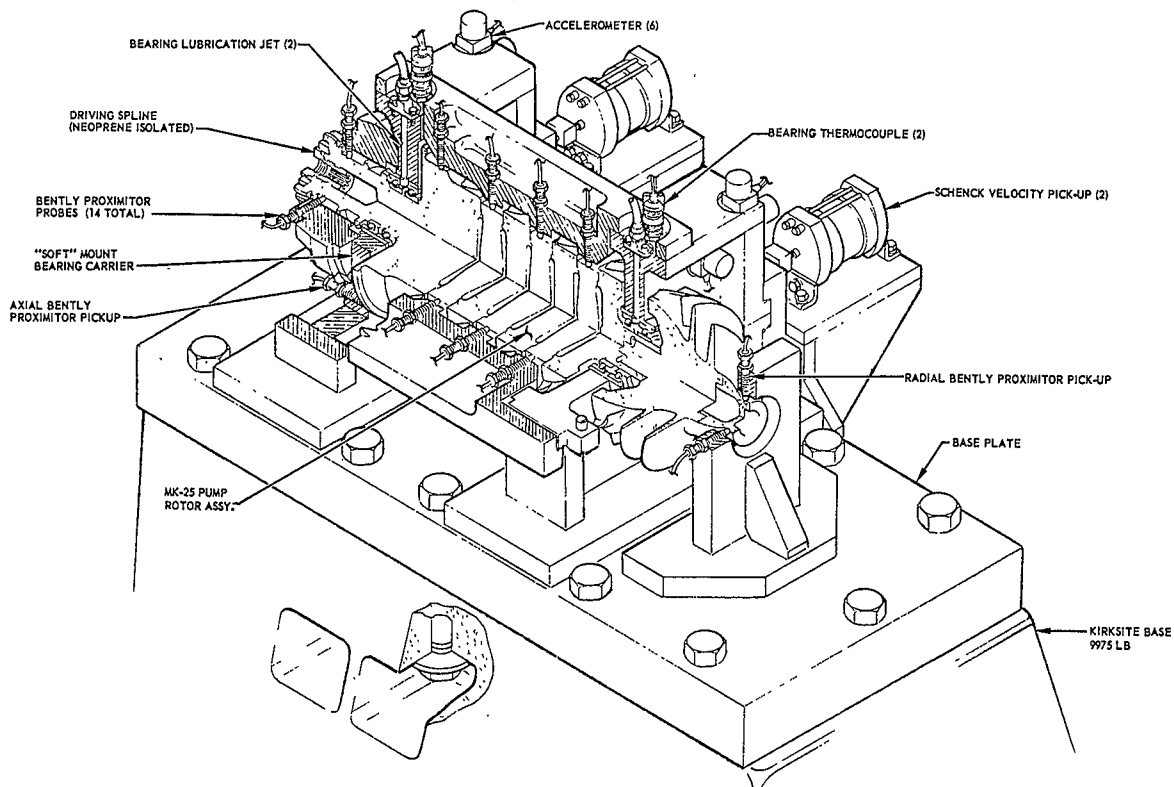


Figure 9. Mark-25 Rotor High Speed Test Set-Up

TEST SET-UP

Hardware

The test set-up consisted of a Mark 25 pump rotor mounted in a support pedestal which was bolted to a large seismic mass. The complete assembly was mounted inside a large vacuum chamber. The pump bearings were cooled by a Freon 21 system. Rotating speeds up to 60,000 rpm were provided by a prime mover (300 hp dynamometer) and a 10:1 gear ratio speed increaser. The rotor was driven by means of an aluminum quill shaft and a splined adapter with a neoprene isolator to minimize transmission of gear noises.

Instrumentation

The basic instrumentation consisted of the following:

- 1 channel, rotor speed
- 14 channels, rotor displacements
- 2 channels, pedestal vibration
- 2 channels, bearing temperatures

The primary instrumentation parameters were the rotor displacement measurements which were made with Bently proximity transducers. A typical installation of these transducers is shown in Fig. 9.

Two mounting bars for the displacement transducers were located along the rotor axis in two 90 degree planes and bolted to the bearing pedestals. The fourteen non-contacting displacement indicators were used to monitor the radial deflection of the rotor in six radial planes and the axial position of the rotor. A calibration spot face was used to determine actual displacement values during test runs.

TEST PROGRAM

The test program was accomplished in two parts. The first part was a set of spin tests to establish an initial balancing condition and the second part was to perform spin tests to investigate unbalance conditions. These unbalance conditions would produce response data for correlation with the mathematical computer model results.

A total of 16 tests were performed as part of this contract study. The first three tests were performed without the inducer. The next three were performed with the inducer. The remaining 10 tests were investigative. One of the investigative tests, 1140, was not used because of loss of data due to a tape recorder malfunction.

Balancing Tests

Three tests (1130 through 1132) were performed without the inducer to obtain an initial balance on the basic rotor. The inducer was then installed and test 1133 performed. During test 1133, the maximum displacement at the forward (inducer) end was .0174 inches at 26,000 rpm. A Post-test examination revealed an inadequate inducer pilot depth. The pilot depth was increased and tests 1134 and 1135 were performed. The maximum displacement for test 1134 was .0113 at 28,000 rpm, but was only .0051 at 26,000 rpm which indicated a significant balance improvement over test 1133. Figure 10 presents the inducer forward displacement for tests 1133 and 1134. Test 1135 was an attempt to achieve an improved balance but was terminated at 24,000 rpm due to excessive deflections.

Investigative Tests

The eight investigative tests, 1136 through 1139 inclusive, and 1142

MK-25 ROTORDYNAMIC DATA
 INDUCER FORWARD END
 DISPLACEMENT VS. ANGLE

○ TEST 1133
 □ TEST 1134

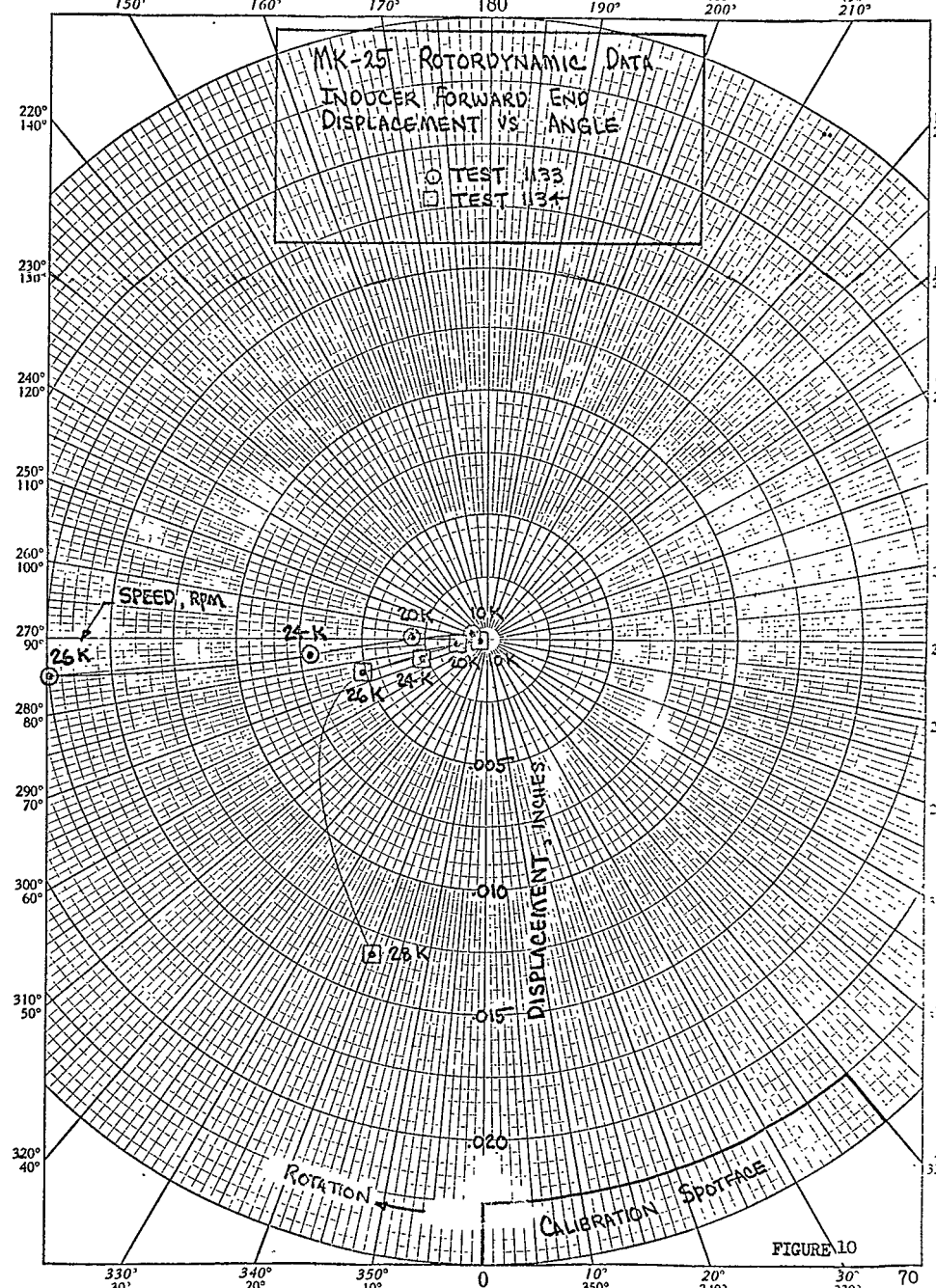


FIGURE 10

through 1145 were performed with various unbalance conditions. Test 1141 was selected as the reference run since it was the best balance achieved. Response deflections of the rotor for the other tests are given as vector changes from the deflections observed in 1141. The forward inducer end and the coupling end displacements for test 1141 are shown in Figure 11. In order to include the effect of critical speed only the tests that operated up to 30,000 rpm or greater were used to generate investigative data. The critical speed was calculated to be approximately 28,000 rpm.

Table 3 lists the test run, the unbalance condition and the upper speed limit achieved. The unbalance weights were selected to allow operation to at least 30,000 rpm without creating excessive bearing loads or allowing a deterioration problem to develop.

PROCEDURE FOR TEST SET-UP

Prior to installation in the rotordynamics test facility, the pump rotor assembly was dynamically balanced at low speed (1000 rpm) on the Gisholt balancing machine. The balancing sequence was as follows:

1. All individual rotors and rotor spacers were dynamic balanced on an arbor.
2. The inducer was balanced on an arbor.
3. The rotors and rotor spacers were assembled into a composite rotor assembly by means of thru bolts, and this assembly mounted in balance bearings was dynamic balanced on the Gisholt machine.
4. The balance bearings were removed, and the rotor was assembled to a pump configuration with duplex bearings, special bearing cartridges, and all rotating parts except the inducer, spinner, and mounting studs and fasteners. Assembled in this manner the rotor was again dynamic balanced. The inducer was then installed on the rotor and this assembly balanced. The spinner was then added and balance corrections made. All components of the rotor were matchmarked.

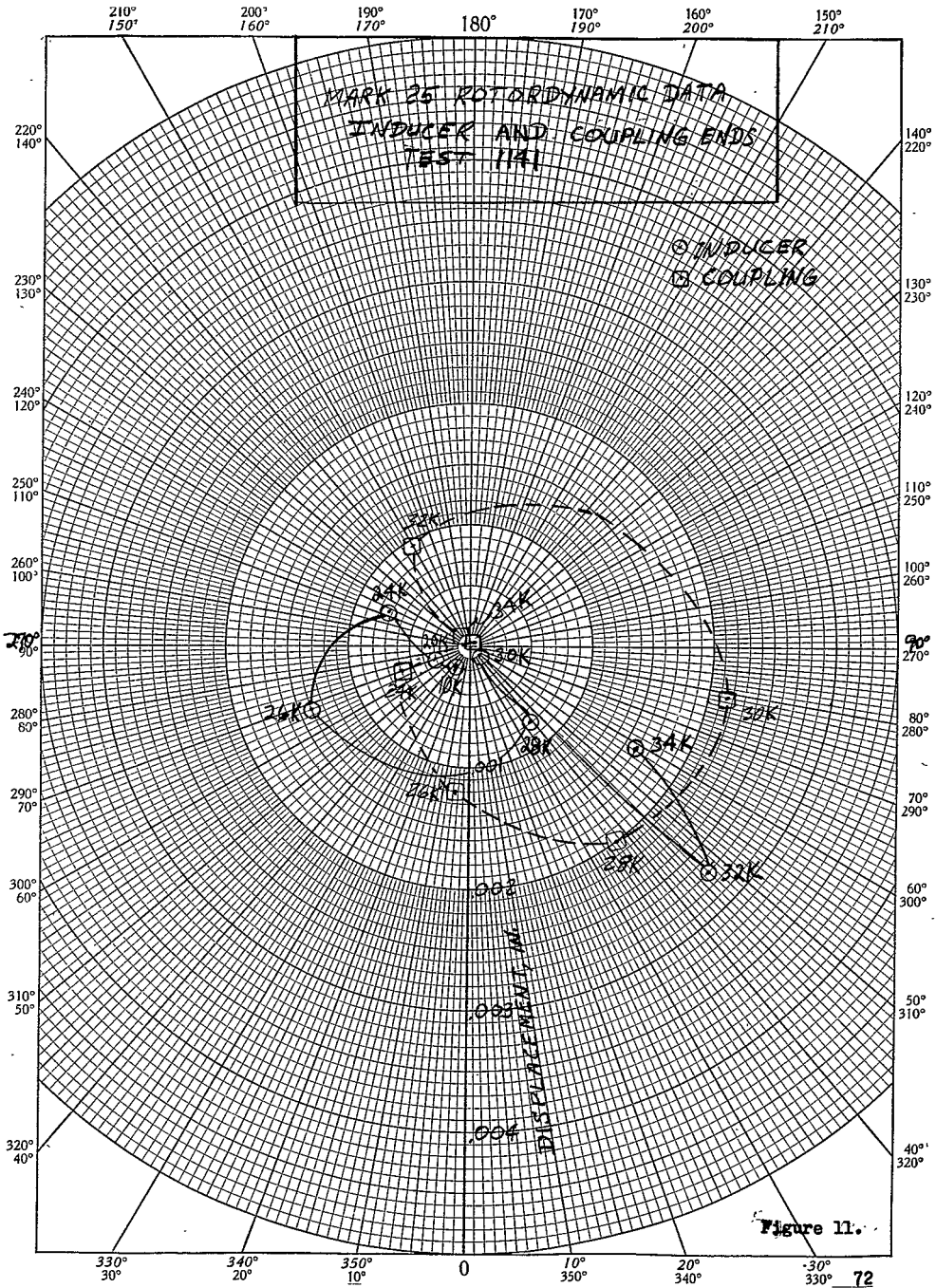


Figure 11.

TABLE 3

UNBALANCE AND UPPER SPEEDS
FOR EXPERIMENTAL TESTS

Test No.	Unbalance *		Location		Speed rpm
	Amount Gram-In	Angle Degrees	Plane No. Shown in Fig. 12	Inches From Inducer	
1136	2.12	138	2	1.8	30,000
	3.40	139	3	5.2	
1137	2.74	195	2	1.8	30,000
	4.25	196	3	5.2	
1138	0.26	-24	2	1.8	34,000
	0.365	-24	3	5.2	
1139	0.64	-32	2	1.8	34,000
	1.10	-30	3	5.2	
1142	1.47	121	10	12.9	32,000
1143	.118	165	9	24.0	30,000
1144	.51	125	7	19.5	30,000
1145	.52	115	4	7.8	30,000

* Reference to Test 1141 and machined spot face on the coupler.

5. The inducer, spinner, and mounting studs were removed from the rotor assembly and the rotor minus the above parts was installed in the cradle mount in the rotordynamics test facility.
6. After the rotor had been installed, all Bently proximitors were gapped, rotor runouts were taken and recorded, and Freon lube jets, bearing thermocouples, and accelerometers were installed.
7. After the rotor was initially balanced, the inducer and spinner were installed so that the complete rotor assembly could be balanced.

ROTORDYNAMICS SPIN TEST PROCEDURE

Prior to the first test each day, the speed increasing gear box was heated to operating temperature by recirculating lube oil through heaters by electrically driven pumps in order to ensure vertical alignment of the center lines on gear box pinion shaft and test rotor. Also, all electronic equipment, oscilloscopes, tape recorders, etc. were turned on to warm up and be checked out. After the above was established, the sequence of events was as follows:

1. The lube oil pumps and heaters were turned off, and the lube oil run tank was charged.
2. The test chamber was closed and the vacuum pumps were started.
3. The Freon pressurized lube system for pump rotor bearings was charged.
4. All on-line monitoring instruments were set up and calibrated and the appropriate test parameter calibrations were recorded on tape.
5. All other systems and controls in the control room and test cell were made ready for test.

6. When chamber pressure reached approximately 40 mm Hg. pressure, the Freon-21 bearing lube system was turned on while rotating the pump rotor slowly to chill the bearings, rotor and cradle to a stabilized operating temperature.
7. When the pump rotor had been chilled, the Freon flow was stopped, bearing purge was turned on, and all Bently detector potentiometers were set for proper voltage output.
8. The test area was cleared and all stations in the control room were manned. The bearing purges were turned off; lube oil and Freon lube systems were turned on and flows were established.
9. The tape recorders and Brush recorders were turned on.
10. The test was started and rotor speed was manually programmed to follow a speed ramp run schedule with a series of speed plateaus. In addition to recording test parameters, one channel on each tape recorder was used to monitor voice, and rotor speed was announced in approximate intervals of five seconds as well as when speed was being changed. This channel also monitored voice communication between the control room and data acquisition room.
11. During the test, Bently output signals were monitored on four two-channel oscilloscopes.
12. At the completion of a test, vacuum was broken in the test chamber, and the tape played back.
13. Phase angle in degrees and rotor motion in mils for each speed plateau was determined from Polaroid photographs and oscillograph records.
14. The rotor quill shaft was disengaged so that the rotor could be turned, by hand, to check bearings.
15. The appropriate balance corrections or unbalance conditions were made on the rotor assembly.

16. The above procedure was repeated until all required tests were performed.

RESULTS

The test results of the experimental program are given in radial deflection of the rotor at six axial stations along the rotor. The experimental results are presented in graphical CRT plots. There are two types of data presentations for each test:

1. deflections vs speed for a given station,
example: Figure 13
2. deflection vs station for a given speed,
example: Figure 21

For the first type of plot, the stations selected were stations 2 (Bently 16), the inducer end; station 9 (Bently 8), the center of the rotor; and station 17 (Bently 2), the coupler end of the pump. The second type of plot was selected at or near speeds of maximum deflection at one or more stations. Three speeds are shown for each test. The total maximum deflection and the X and Y plane components are shown relative to the spot face at station 17, which was the zero degree (0°) reference point on the rotor. Table 4 gives the relationship between the experimental and mathematical station descriptors.

In order to adjust the data to a form easily used for comparative purposes, the data was corrected to remove the effects of shaft runout and the unknown residual unbalance. The latter was accomplished by vectorily subtracting the data from the minimum response run (the reference run, 1141) from all other test runs. The unbalance condition listed in Table 3 is the "delta" unbalance between the test run and the minimum response run (1141). All angles referred to in the Table 3 and the graphical

TABLE 4

RELATIVE MATHEMATICAL AND EXPERIMENTAL
MODEL DESCRIPTORS

Bently Meas. Location	Math. Model Station	Rotor Axial Position (Ref. Inducer end) Inches
16	2	0.8
10	8	8.75
8	9	11.5
6	11	14.5
4	13	17.25
2	17	22.9

plots are referenced to a spot face at plane 8 (Bently #2), Fig. 12

On the experimental data plots the following nomenclature is used:

- DEFL 2 = deflection of the rotor at Bently #2 (Math Model Sta. 17) vs speed
- DEFL 8 = deflection of the rotor at Bently #8 (Math Model Sta.9) vs speed
- DEFL 16 = deflection of the rotor at Bently #16 (Math Model Sta. 2) vs speed
- X PLANE = deflection of the rotor vs station at indicated speed in the X-plane with reference to the spot face at Station 17.
- Y PLANE = deflection of the rotor vs station at indicated speed in the Y-plane with reference to the spot face at Station 17
- TOTAL = total deflection of the rotor (maximum) vs station

All deflections given are in mils (10^{-3} inches).

Figures 13 to 20, inclusive, are the deflection vs speed plots for test runs 1136 through 1139 and 1142 through 1145, inclusive. The speed range plotted is from 10,000 to 30,000 rpm except for Test 1142 which had an upper speed limit of 32,000 rpm, and Tests 1138 and 1139 which achieved upper speed limits of 34,000 rpm.

Figures 21 to 44, inclusive, are the deflection of the rotor vs rotor stations for three speed cases for each of the investigative tests. The abscissa is the distance from the inducer end in inches. In most cases, the applied unbalance tended to cause the local section of the pump rotor to deflect significantly; i.e., when the unbalance was at the

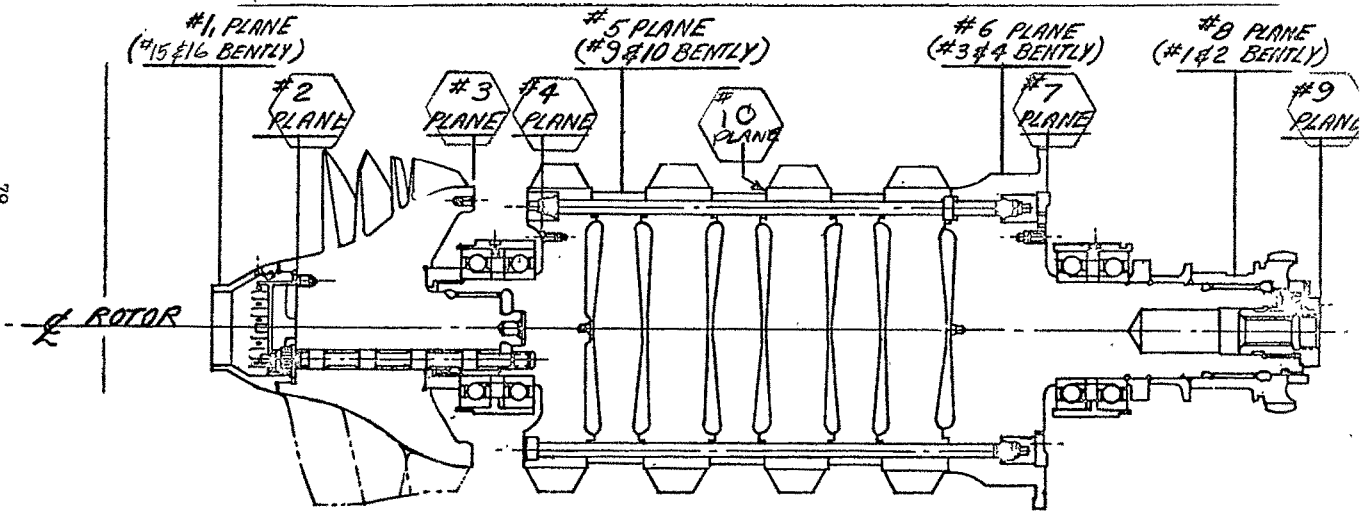


Figure 12. Bently and Balance Planes, Mark-25 Pump Rotor

inducer, the inducer end had the largest deflection; when the unbalance was in the center section (between the bearings) the center of the rotor deflected more. However, when the unbalance (a very small amount) was applied at the coupler end, the inducer displayed large deflections. This was primarily due to the inherent mode shape of the rotor in the speed range of interest.

TEST NO. 1136

BENTLY'S 2, 8 AND 16 DEFLECTIONS (MILS) VS SPEED (RPM)

643953
0005 0000

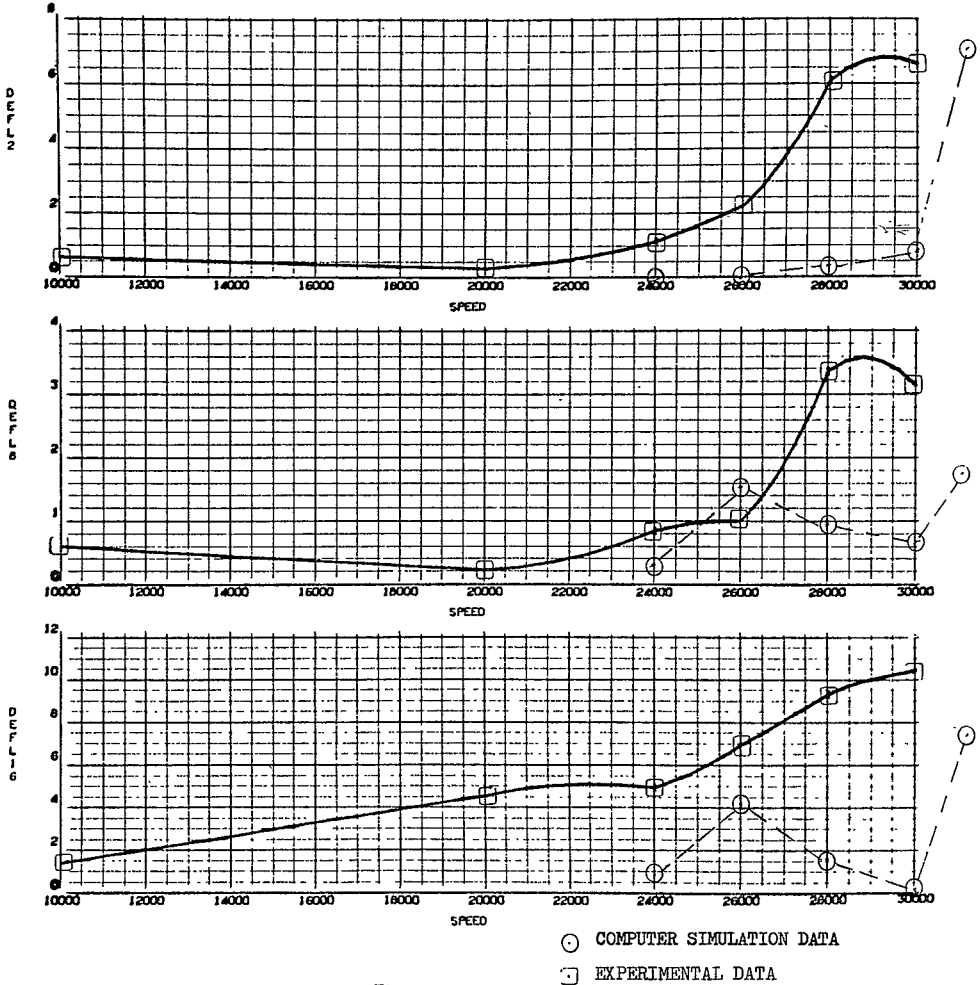
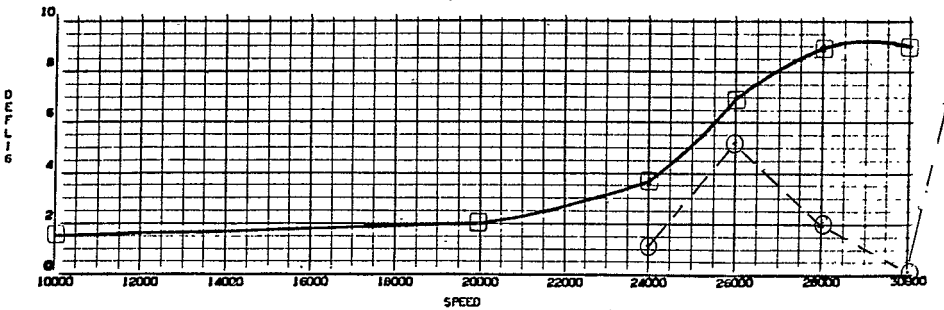
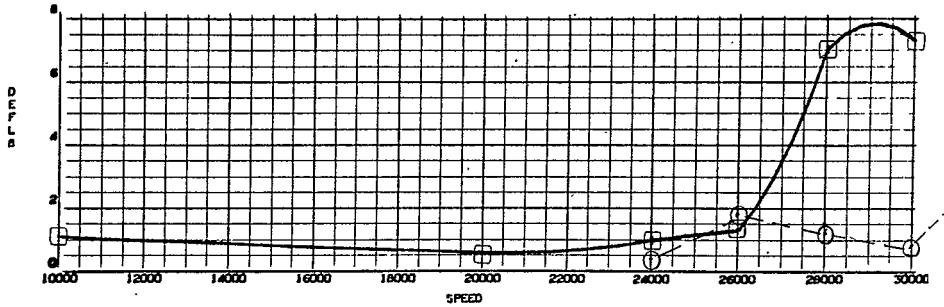
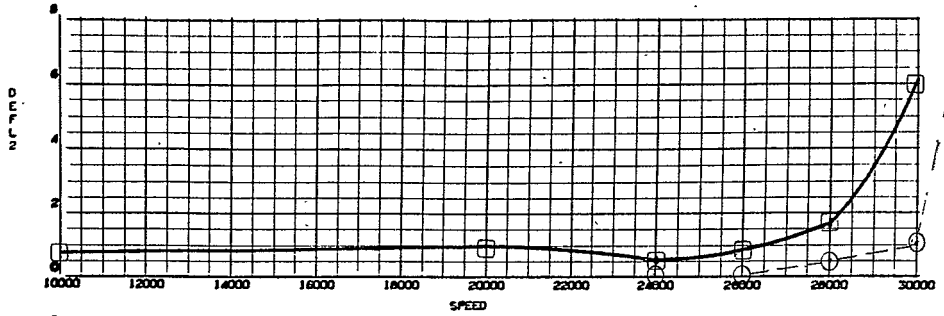


Figure 13

TEST NO. 1137

BENTLY'S 2, 8 AND 16 DEFLECTIONS (MILS) VS SPEED (RPM)

843953
0007 0000



○ COMPUTER SIMULATION DATA

□ EXPERIMENTAL DATA

Figure 14

TEST 1138

BENTLY'S 2, 8 AND 16 DEFLECTION (MILS) VS SPEED (RPM)

847956
0004 0000

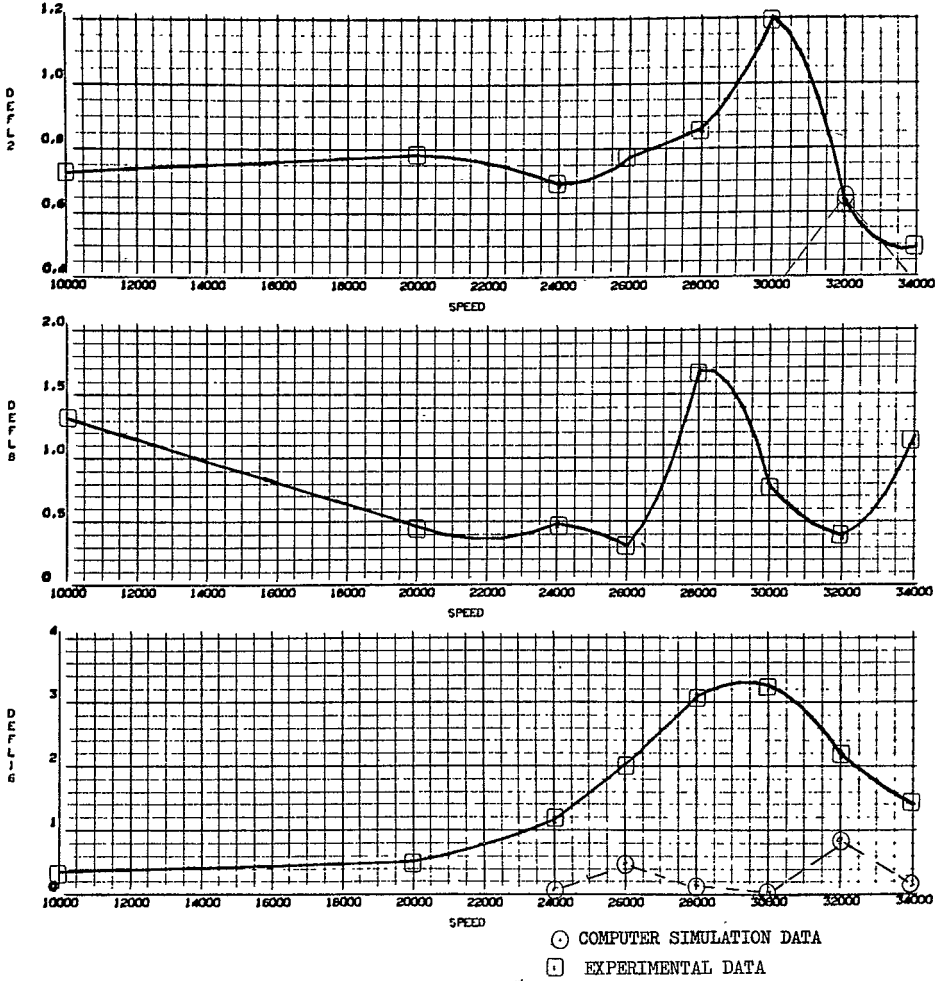


Figure 15

TEST 1139

843956
0001 0000

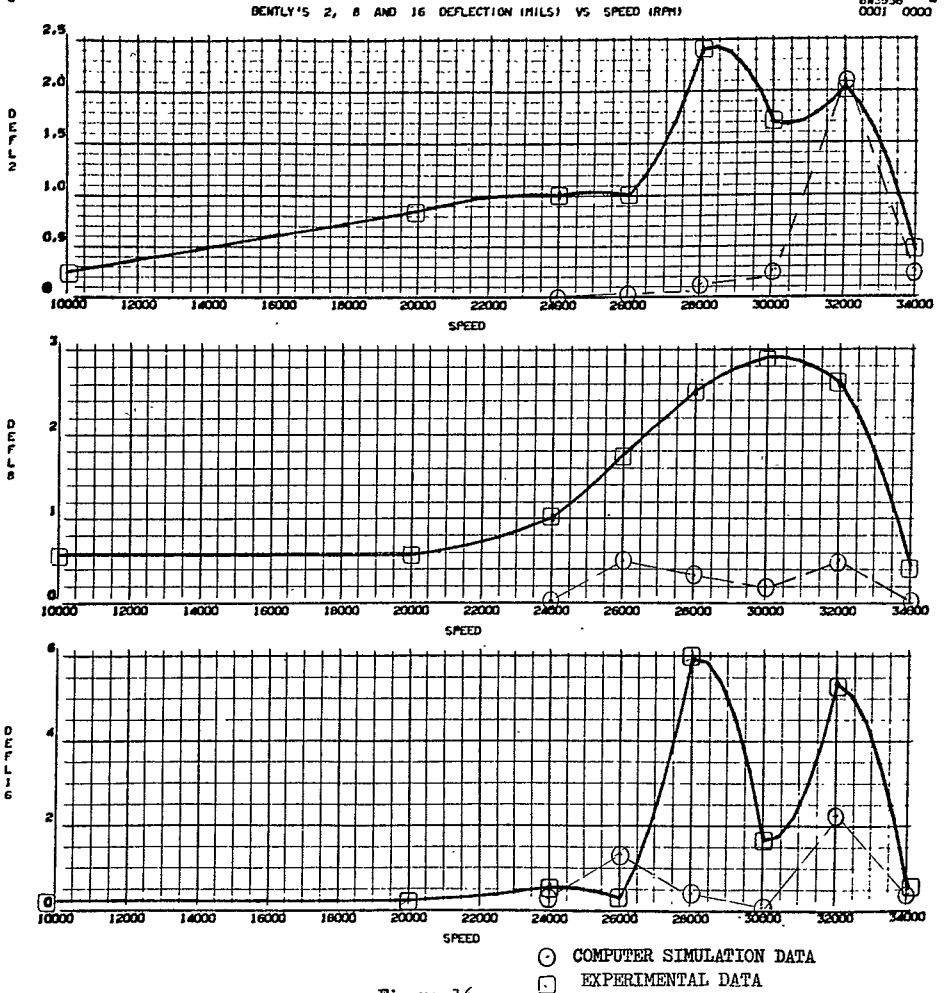


Figure 16

TEST 1142

8439A2
0001 0000

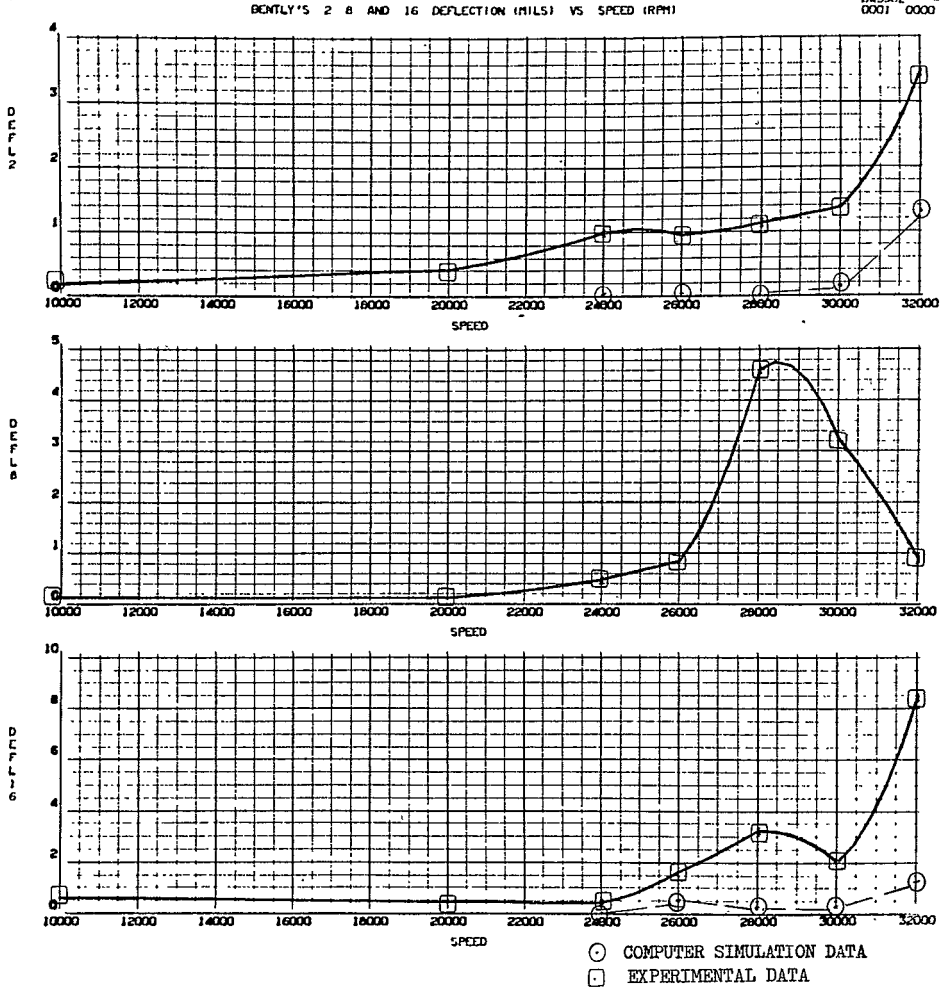


Figure 17

TEST NO. 1143

BENTLY'S 2, 8 AND 16 DEFLECTIONS (MILS) VS SPEED (RPM)

843953
0003 0000

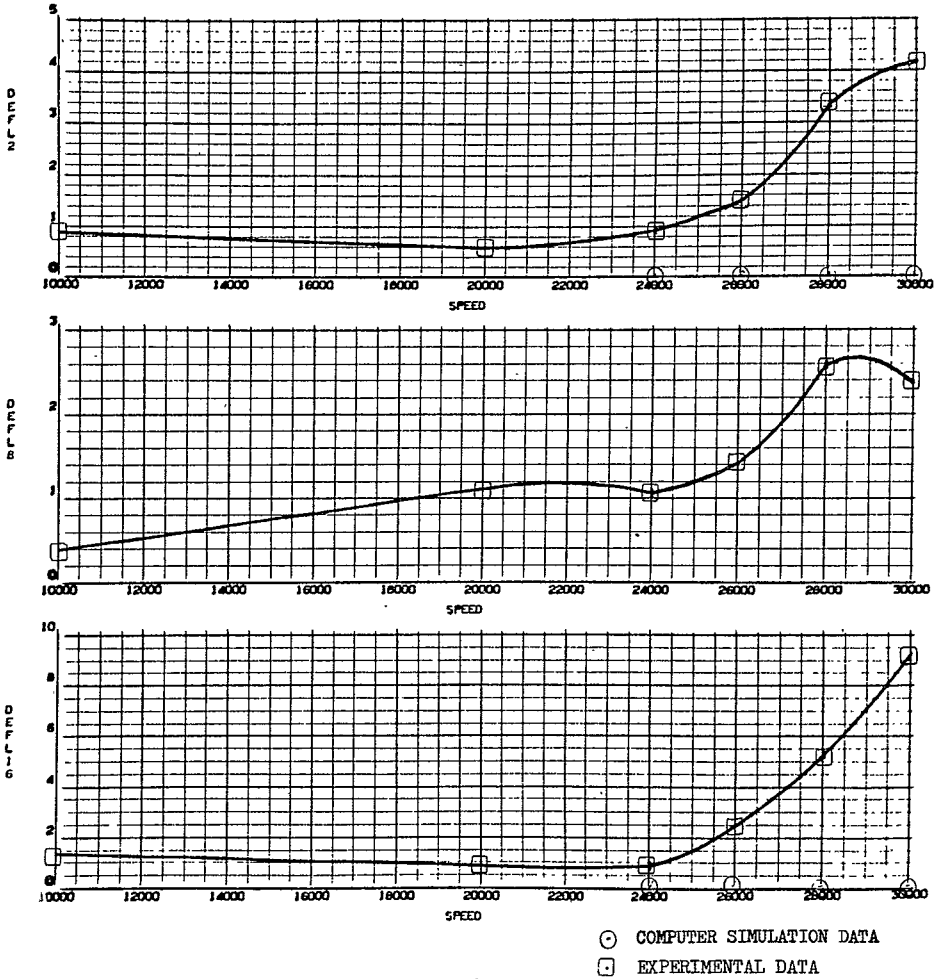


Figure 18

TEST NO. 1144

BENTLY'S 2, 8 AND 16 DEFLECTIONS (MILS) VS SPEED (RPM)

842953
0001 0000

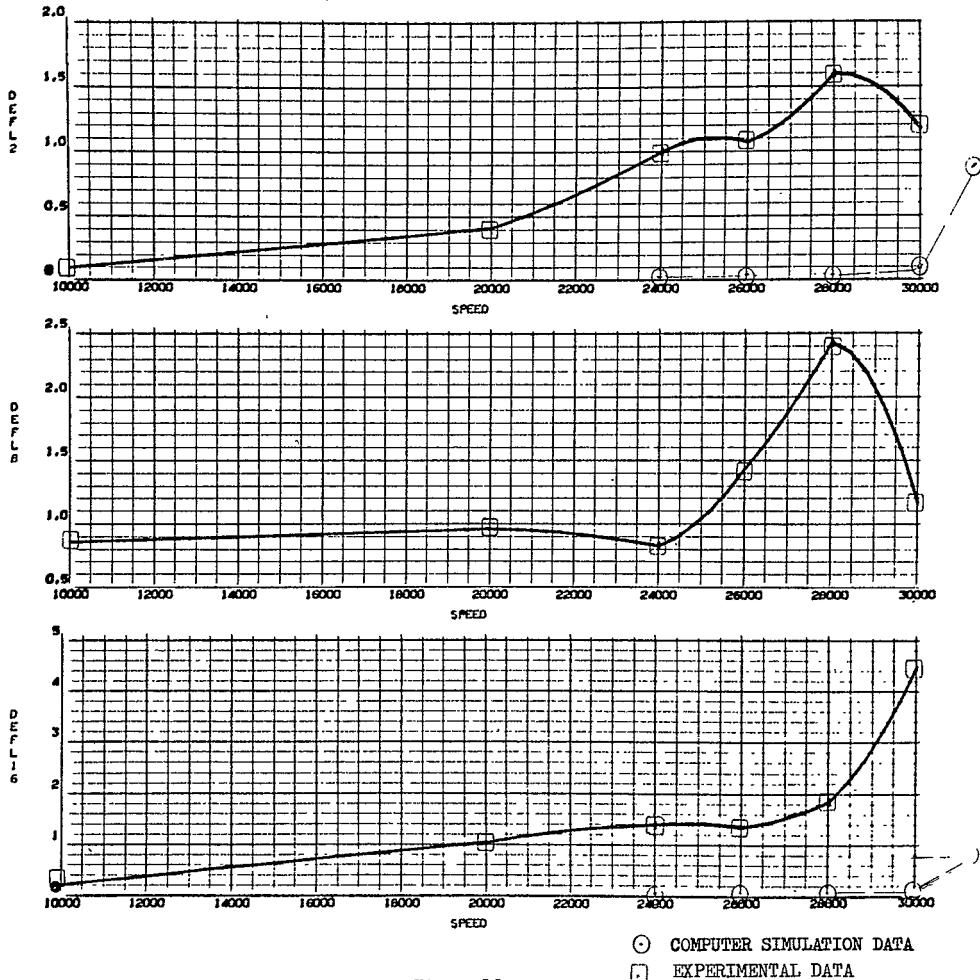
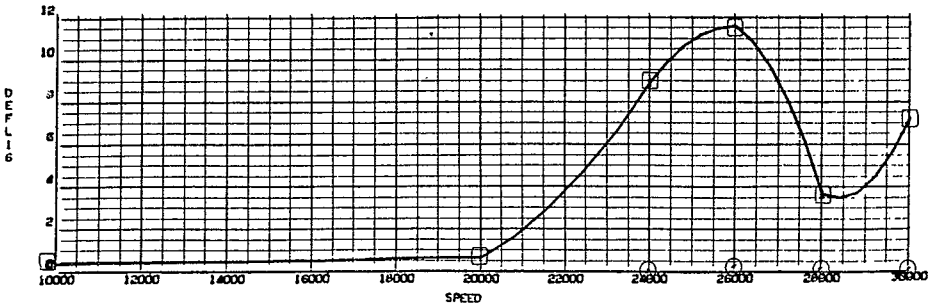
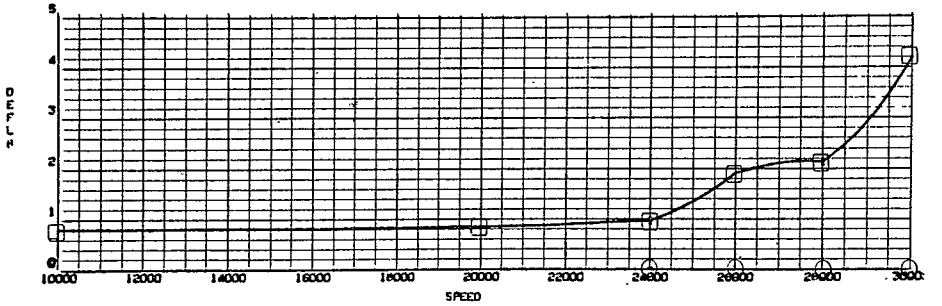
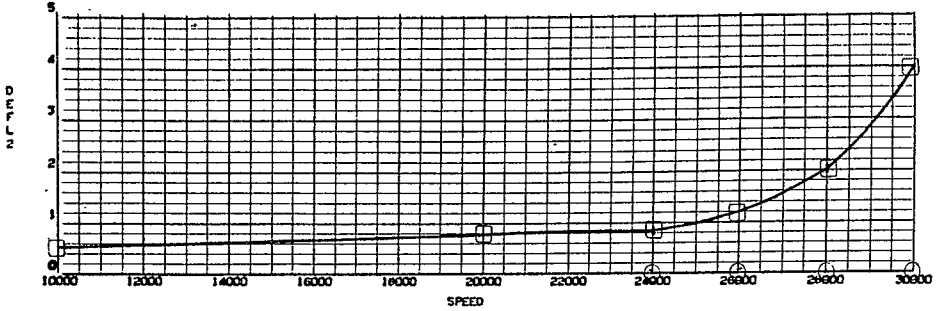


Figure 19

TEST 1145

BENTLY'S 2xH AND 16 DEFLECTIONS (MILS) VS SPEED (RPM)

843944
0001 0000



⊙ COMPUTER SIMULATION DATA
 □ EXPERIMENTAL DATA

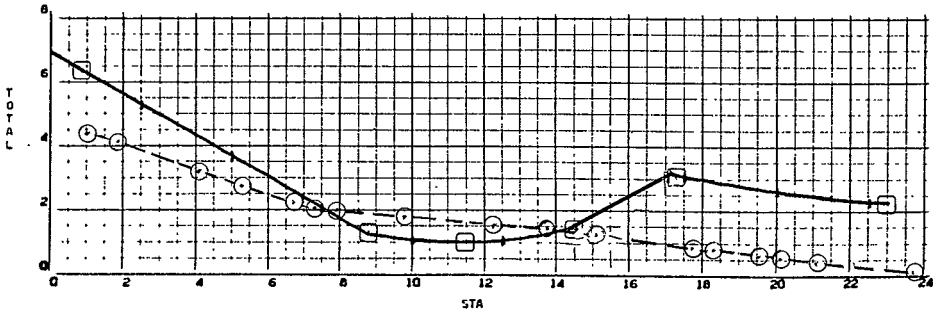
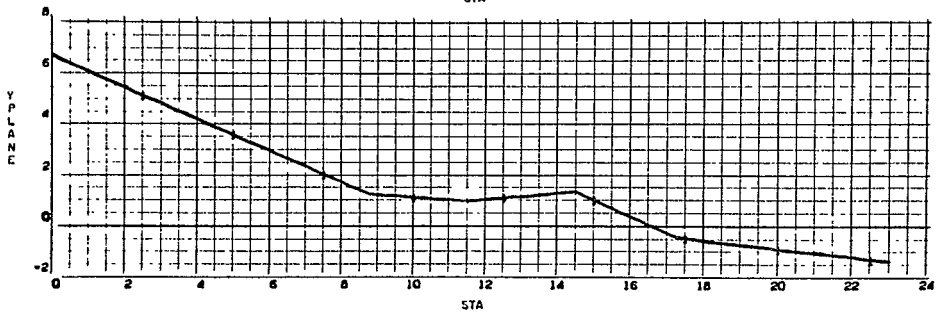
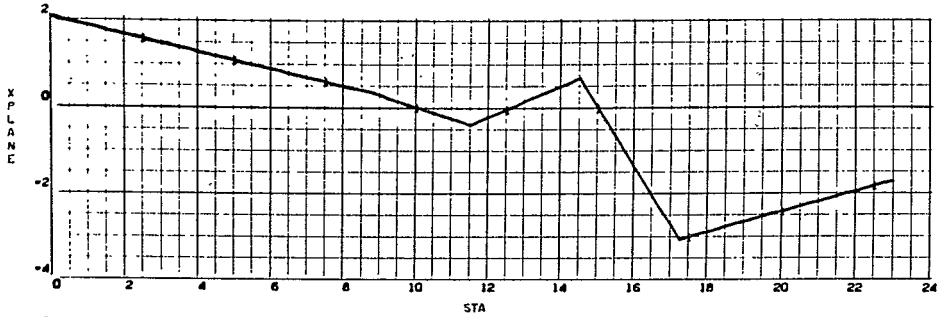
Figure 20

TEST 1136

26000 RPM

A43980
0003 0000

DEFLECTION (MILS) VS STATION (IN)



⊙ COMPUTER SIMULATION DATA

⊠ EXPERIMENTAL DATA

Figure 21

TEST 1136

28000 RPM

843980
0004 0000

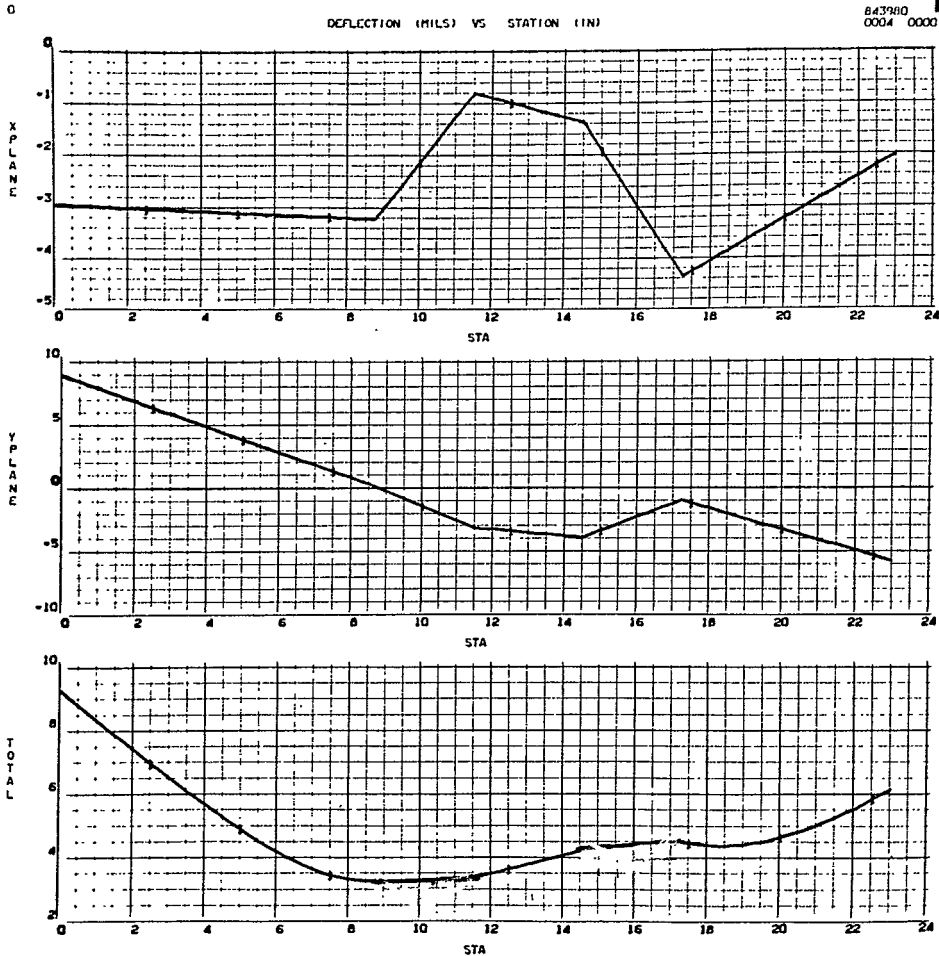


Figure 22

TEST 1136 30000 RPM

A43990
0005 0000

DEFLECTION (MILS) VS STATION (IN)

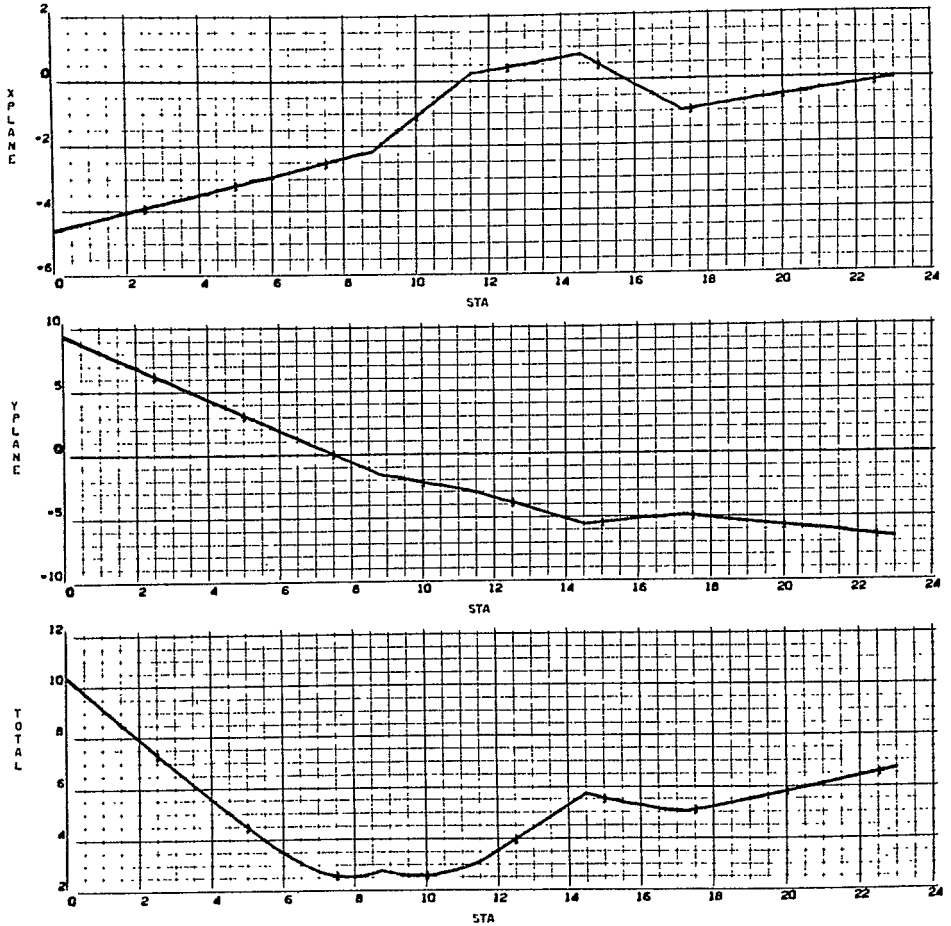


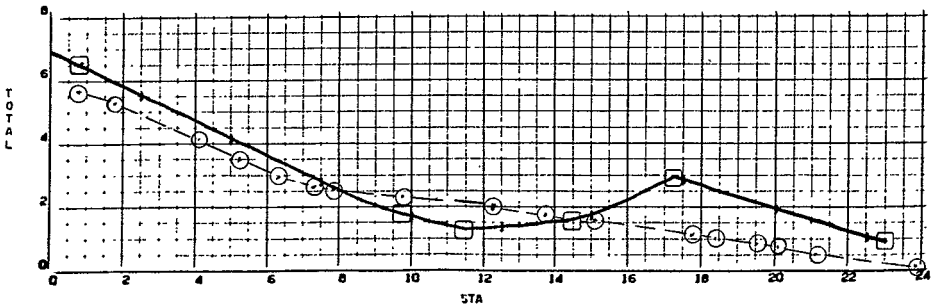
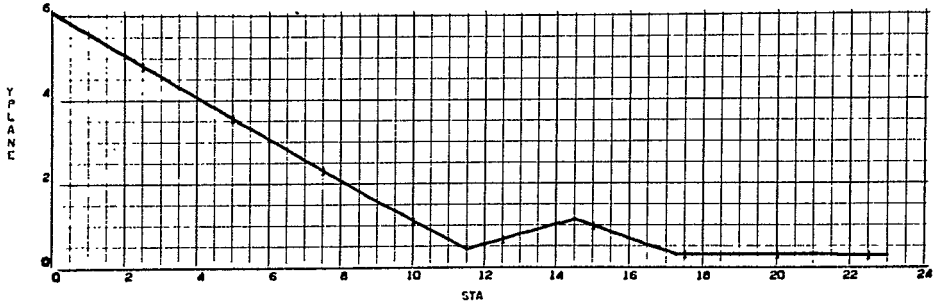
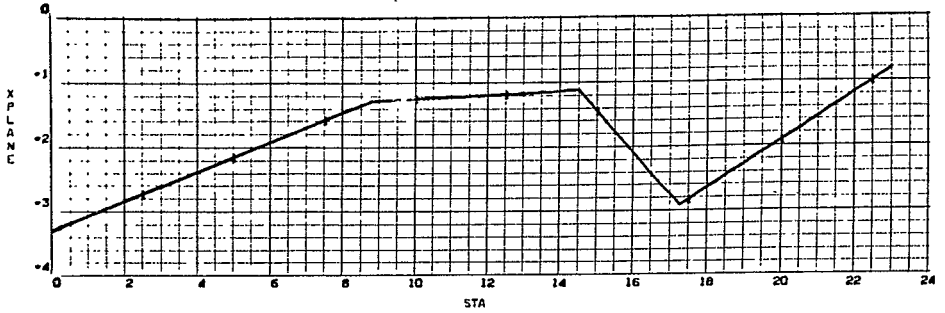
Figure 23

TEST 1137

26000 RPM

843980
0006 0000

DEFLECTION (MILS) VS STATION (IN)



- COMPUTER SIMULATION DATA
- EXPERIMENTAL DATA

Figure 24

TEST 1137

28000 RPM

647980
0007 0000

DEFLECTION (MILLS) VS STATION (IN)

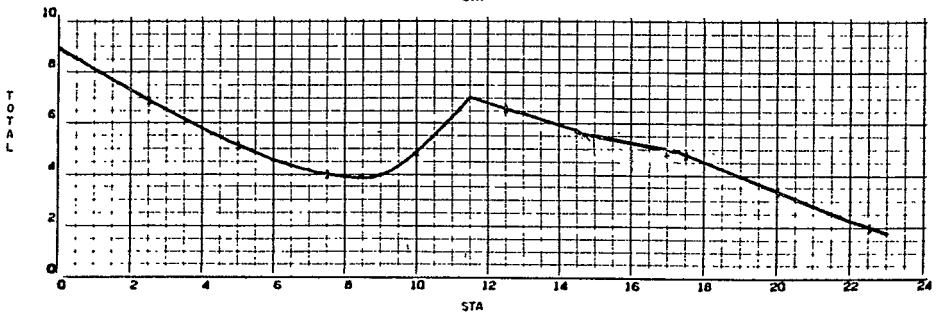
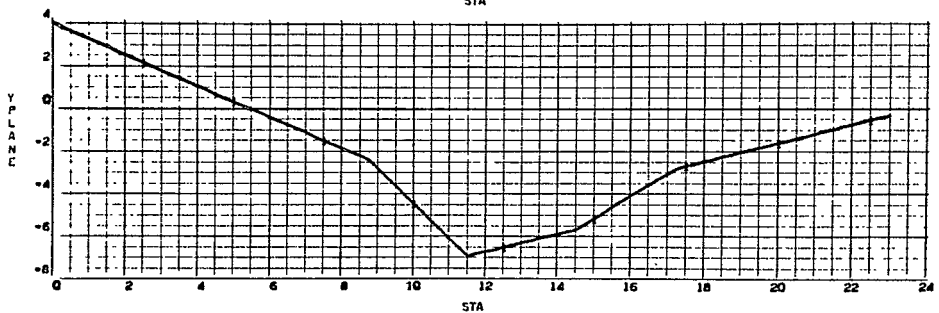
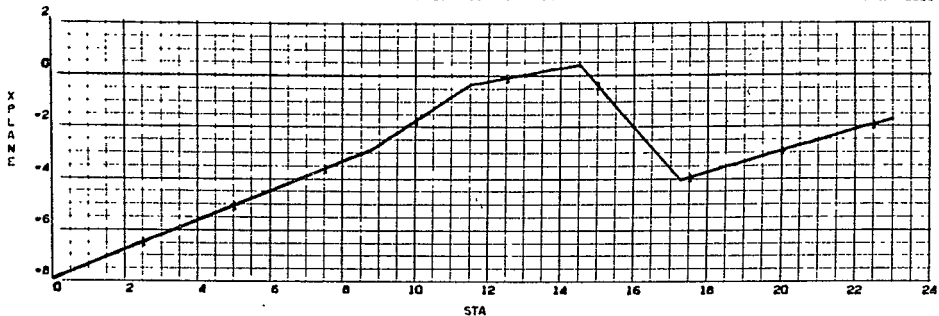


Figure 25

TEST 1137 30000 RPM

DEFLECTION (MILS) VS STATION (IN)

843940
0008 0000

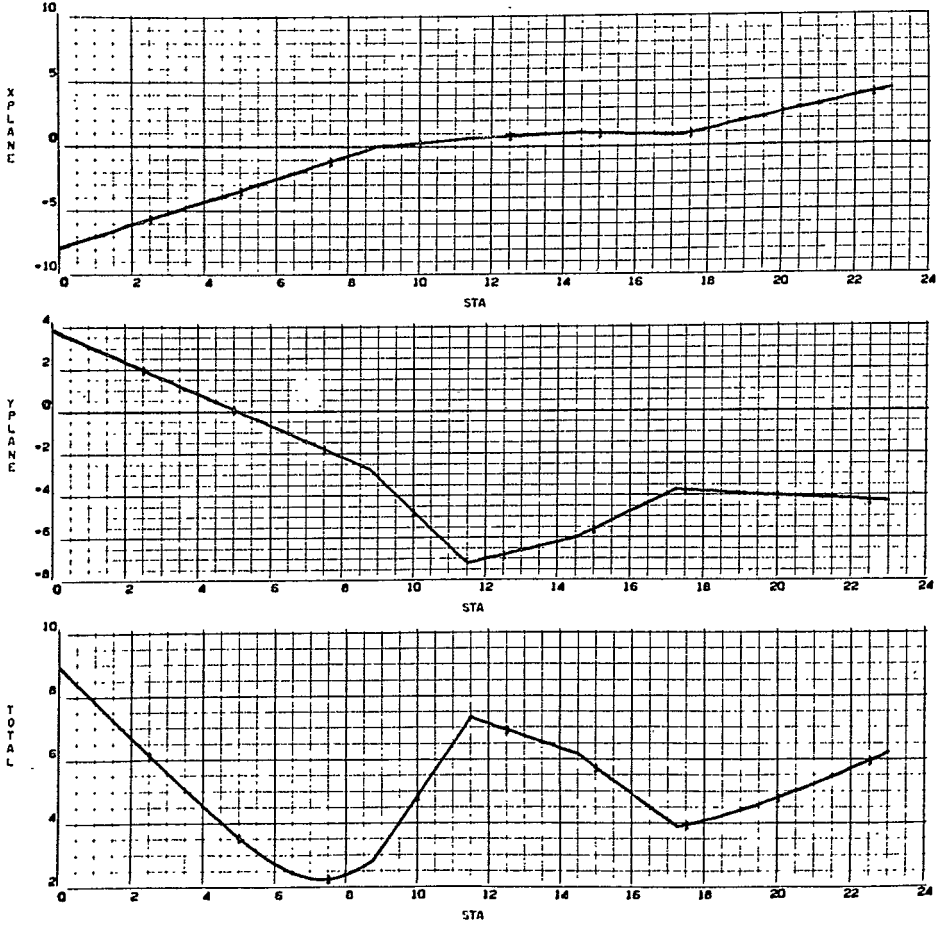


Figure 26

TEST 1138

28000 RPM

B42380
0000 0000

DEFLECTION (MILS) VS STATION (IN)

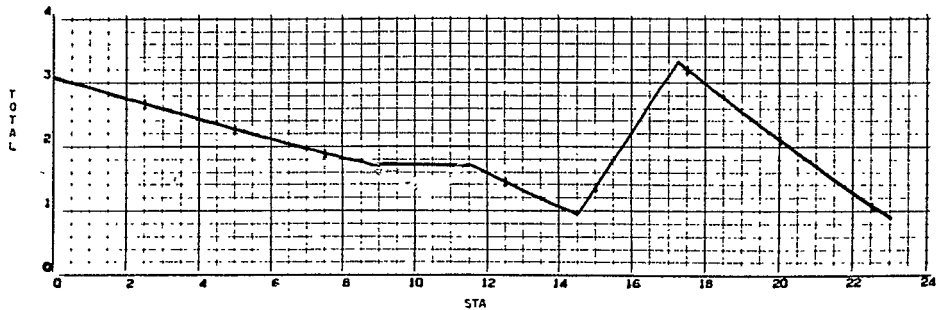
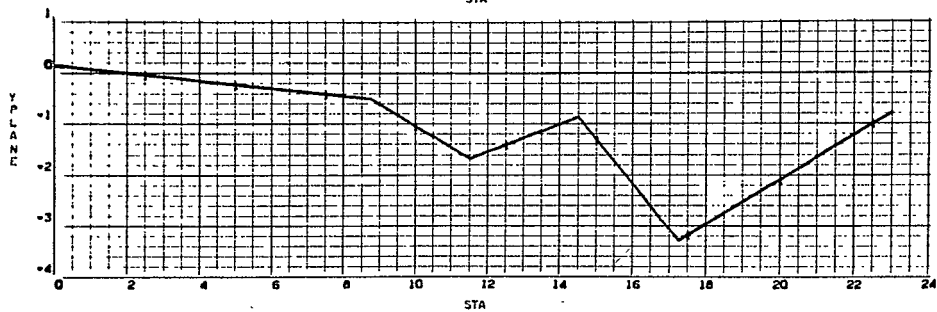
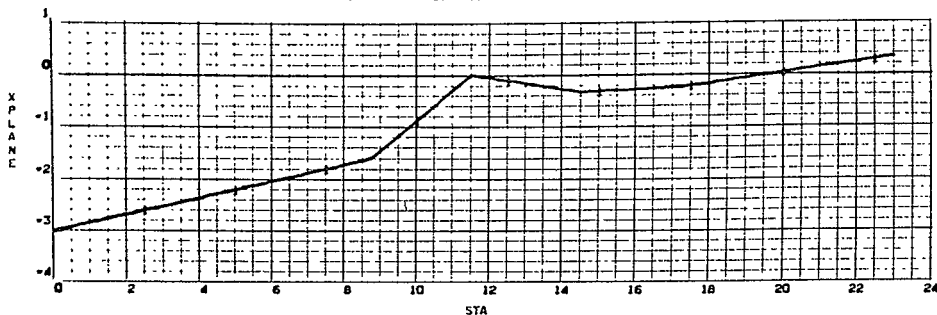


Figure 27

TEST 1138 30000 RPM

BA3980
0010 0000

DEFLECTION (MILS) VS STATION (IN)

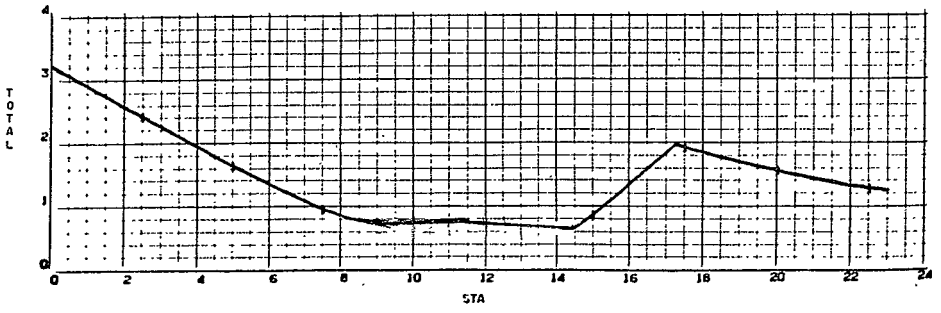
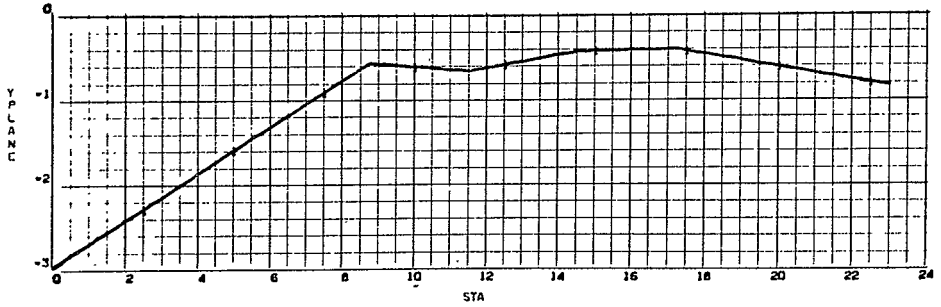
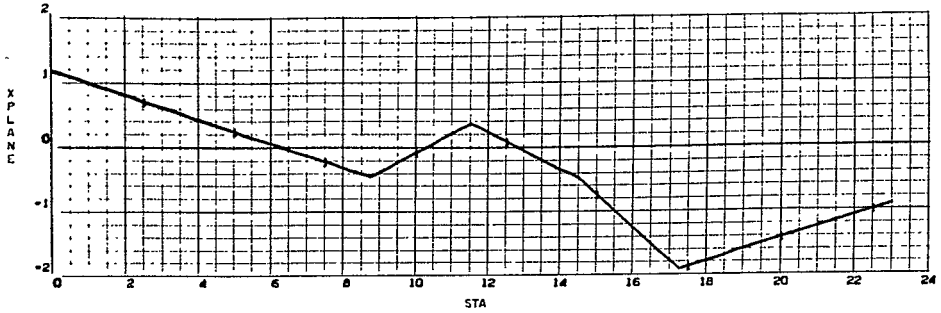


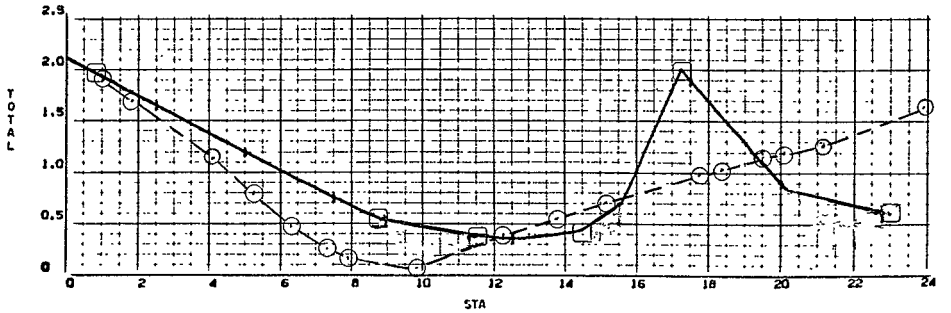
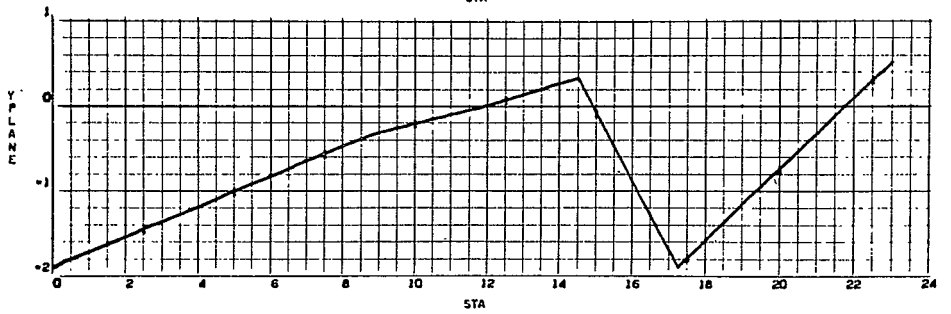
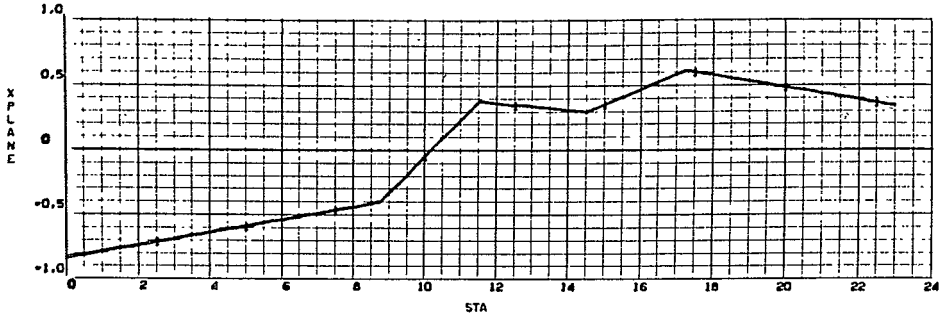
Figure 28

TEST 1138

32000 RPM

842980
0011 0000

DEFLECTION (MILS) VS STATION (IN)



○ COMPUTER SIMULATION DATA (x 2)

□ EXPERIMENTAL DATA

Figure 29

TEST 1139 28000 RPM

843980
0012 0000

DEFLECTION (MILS) VS STATION (IN)

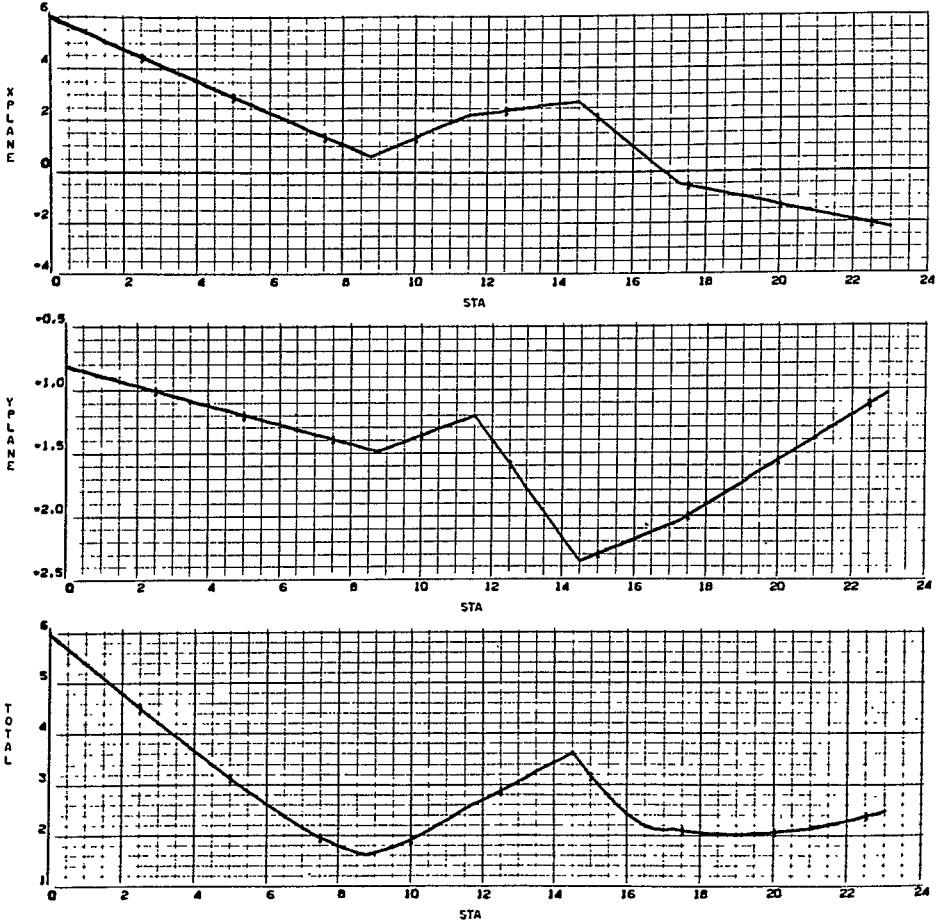


Figure 30

TEST 1139 30000 RPM

843980
0013 0000

DEFLECTION (MILS) VS STATION (IN)

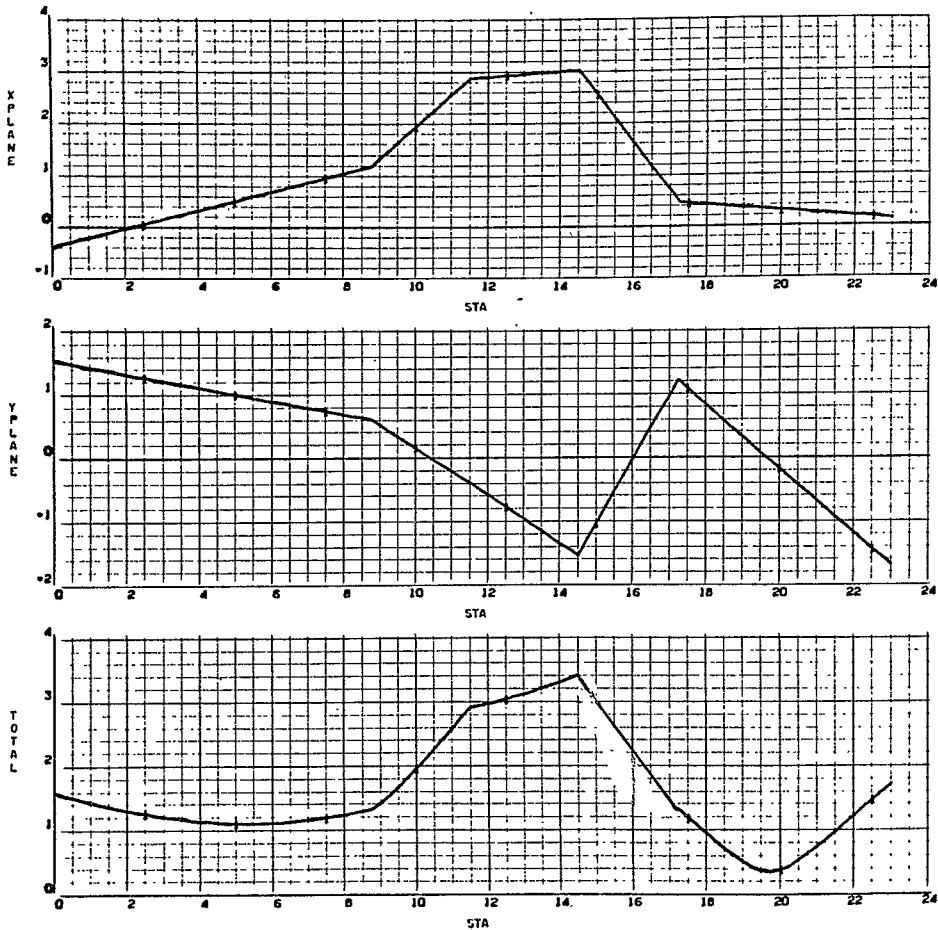


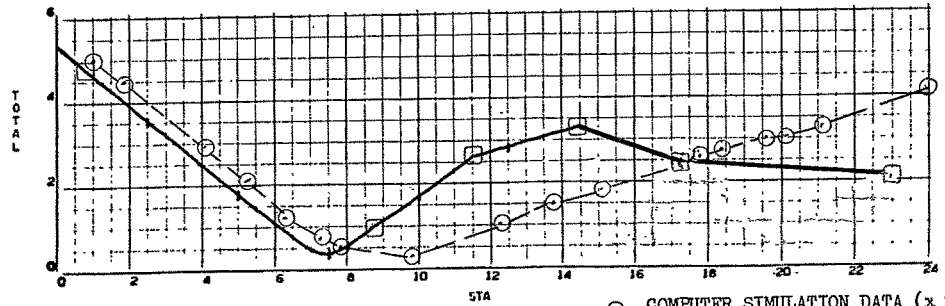
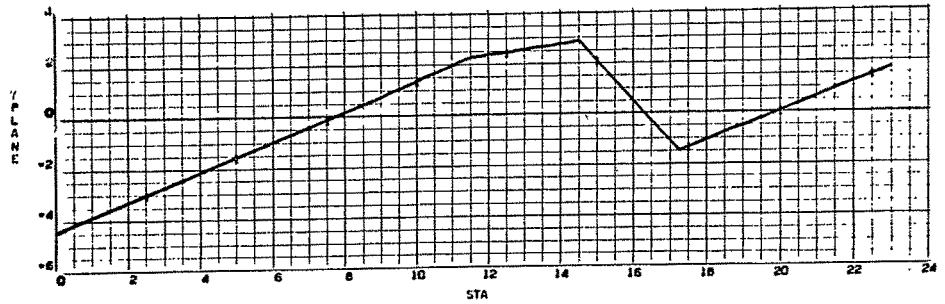
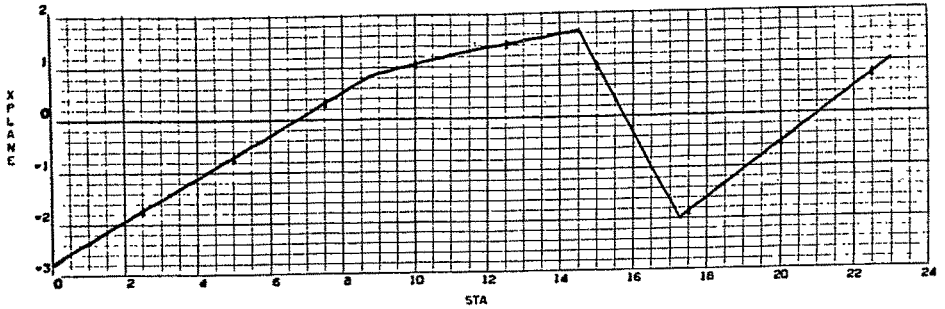
Figure 31

TEST 1139

32000 RPM

843980
0014 0000

DEFLECTION (MILS) VS STATION (IN)



○ COMPUTER SIMULATION DATA (x 2)
□ EXPERIMENTAL DATA

Figure 32

TEST 1142

28000 RPM

843980
0015 0000

DEFLECTION (MILS) VS STATION (IN)

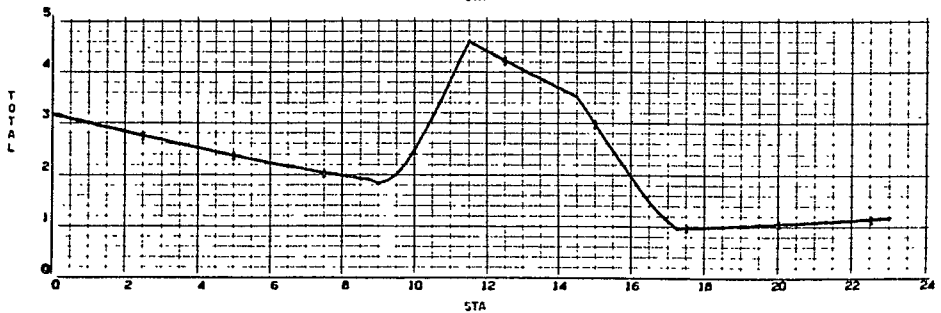
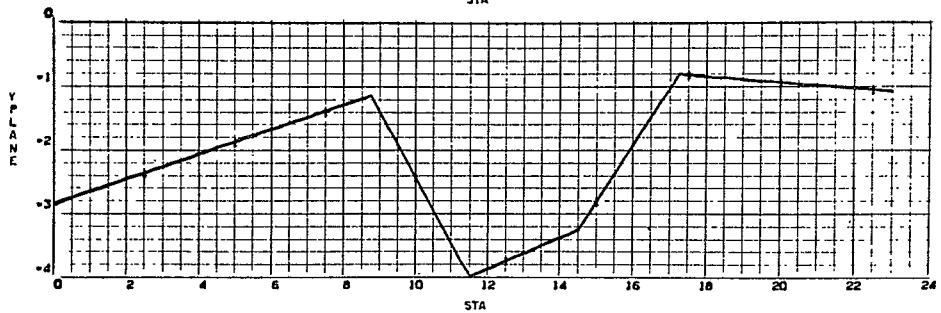
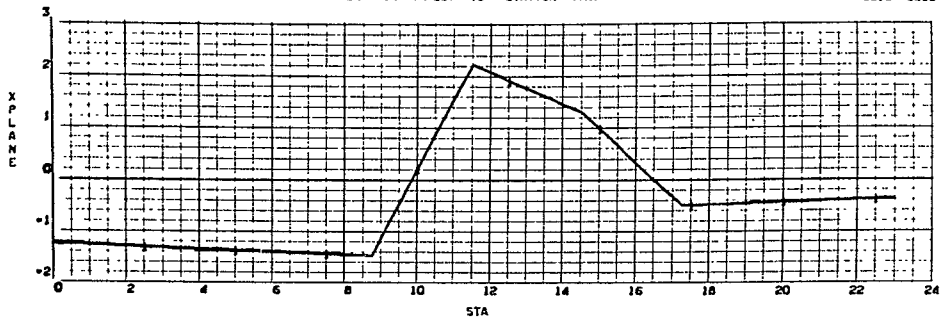


Figure 33

TEST 1142 30000 RPM

B43980
0016 0000

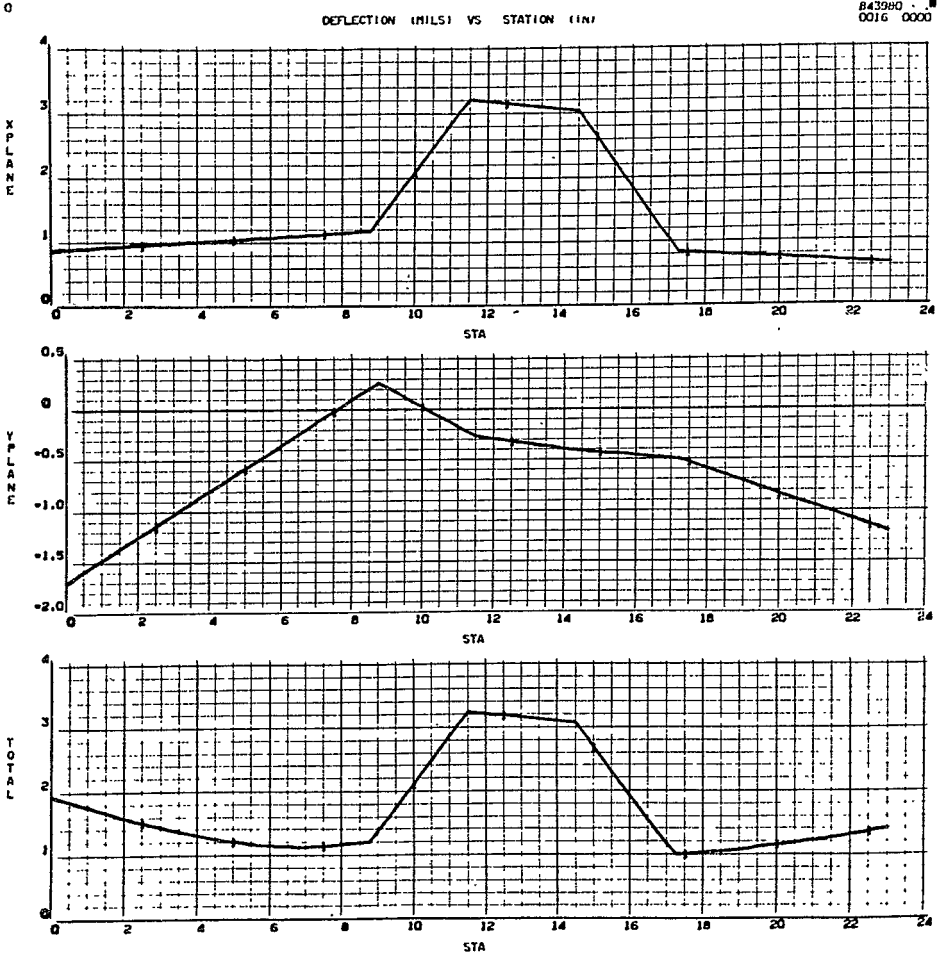
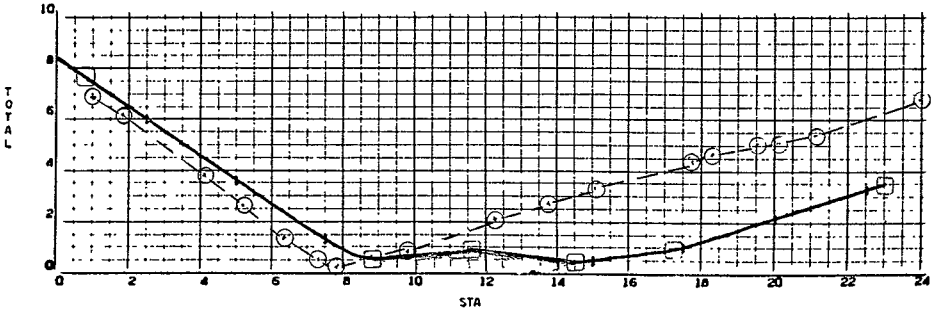
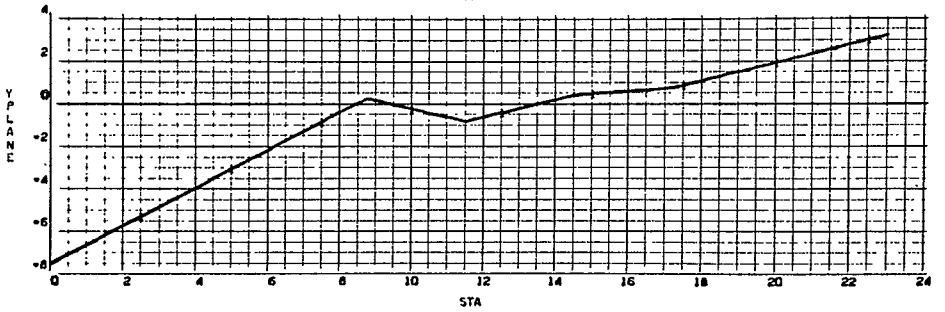
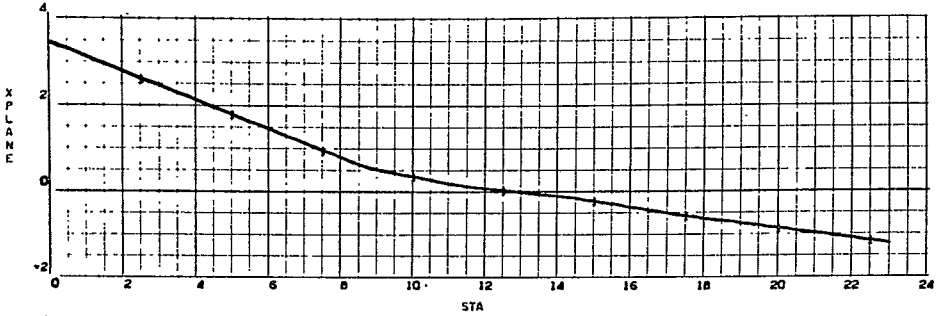


Figure 34

TEST 1142 32000 RPM

047980
0017 0000

DEFLECTION (MILS) VS STATION (IN)



⊙ COMPUTER SIMULATION DATA (x 5)

□ EXPERIMENTAL DATA

Figure 35

TEST 1143 26000 RPM

842900
0022 0000

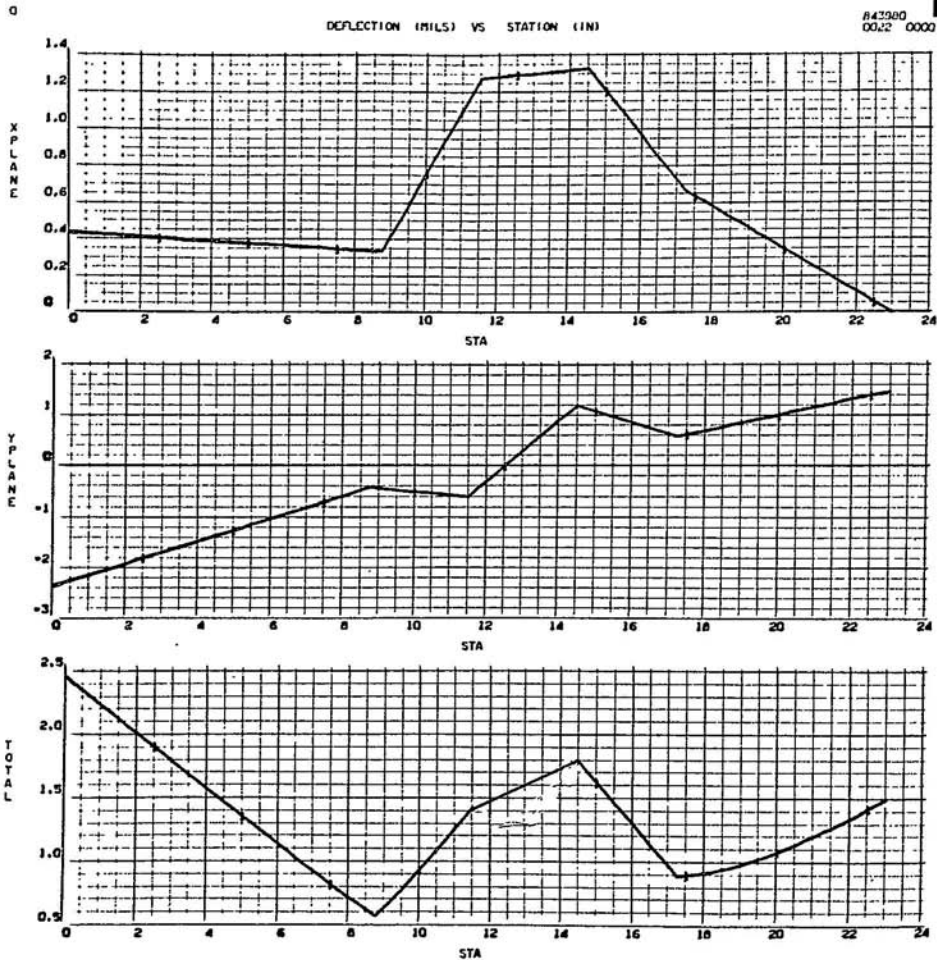


Figure 36

TEST 1143 28000 RPM

643980
0023 0000

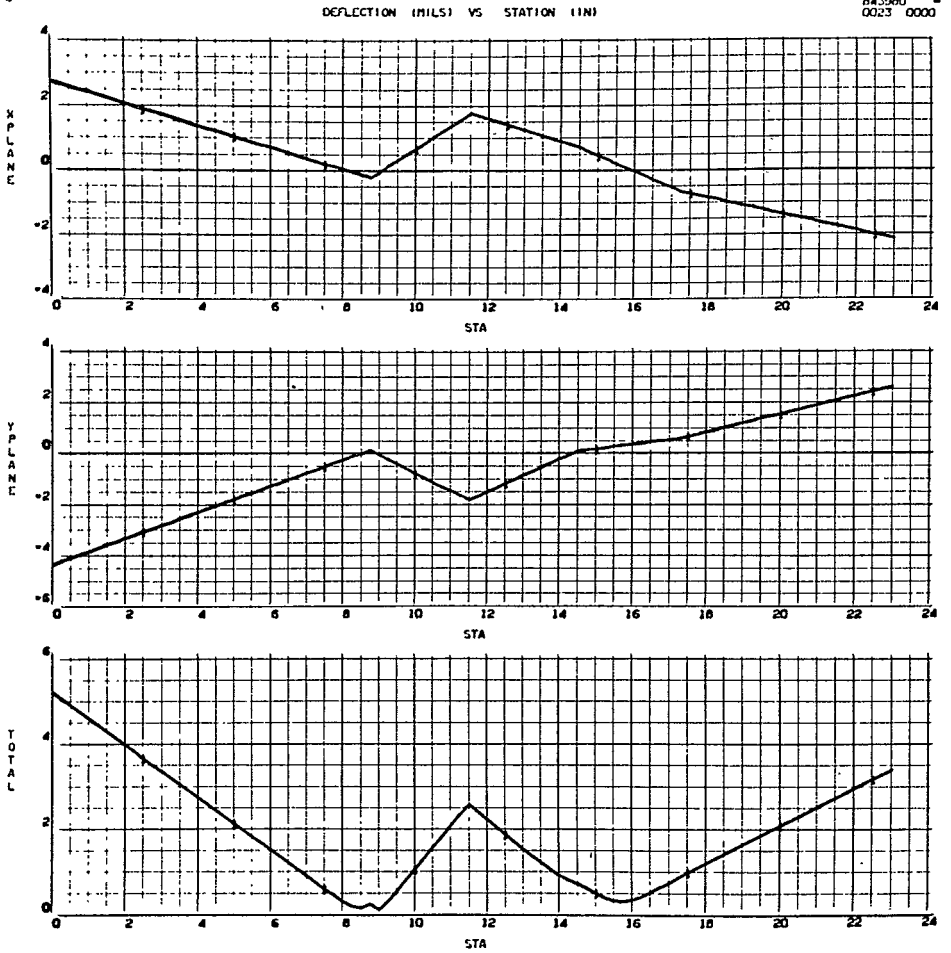


Figure 37

TEST 1143

30000 RPM

643980
0024 0000

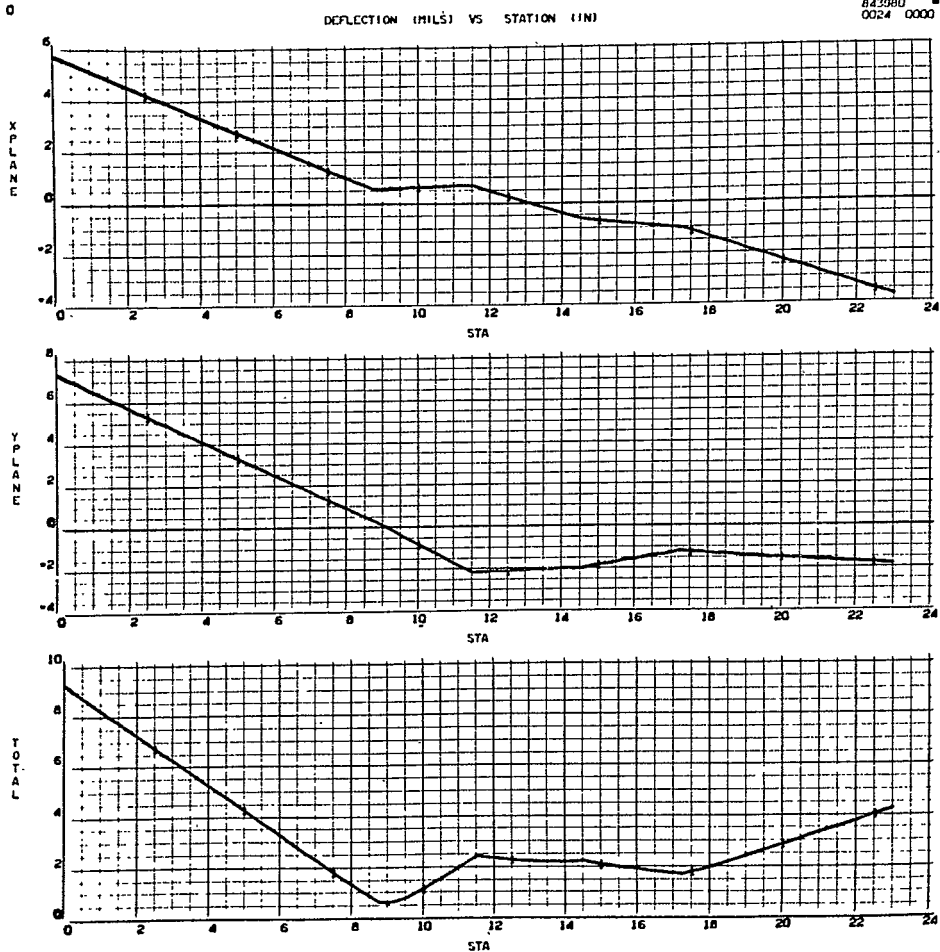


Figure 38

TEST 1144

26000 RPM

8439#0
0018 0000

DEFLECTION (MILS) VS STATION (IN)

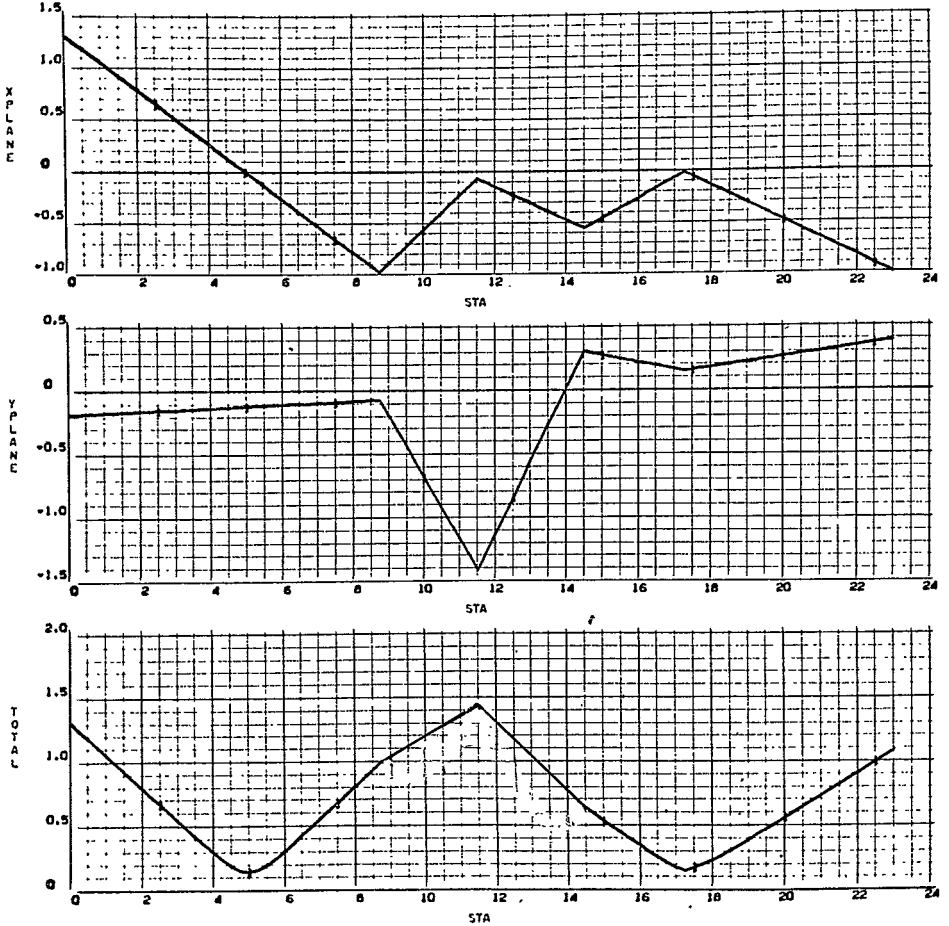


Figure 39

TEST 1144 28000 RPM

#43990
0019 0000

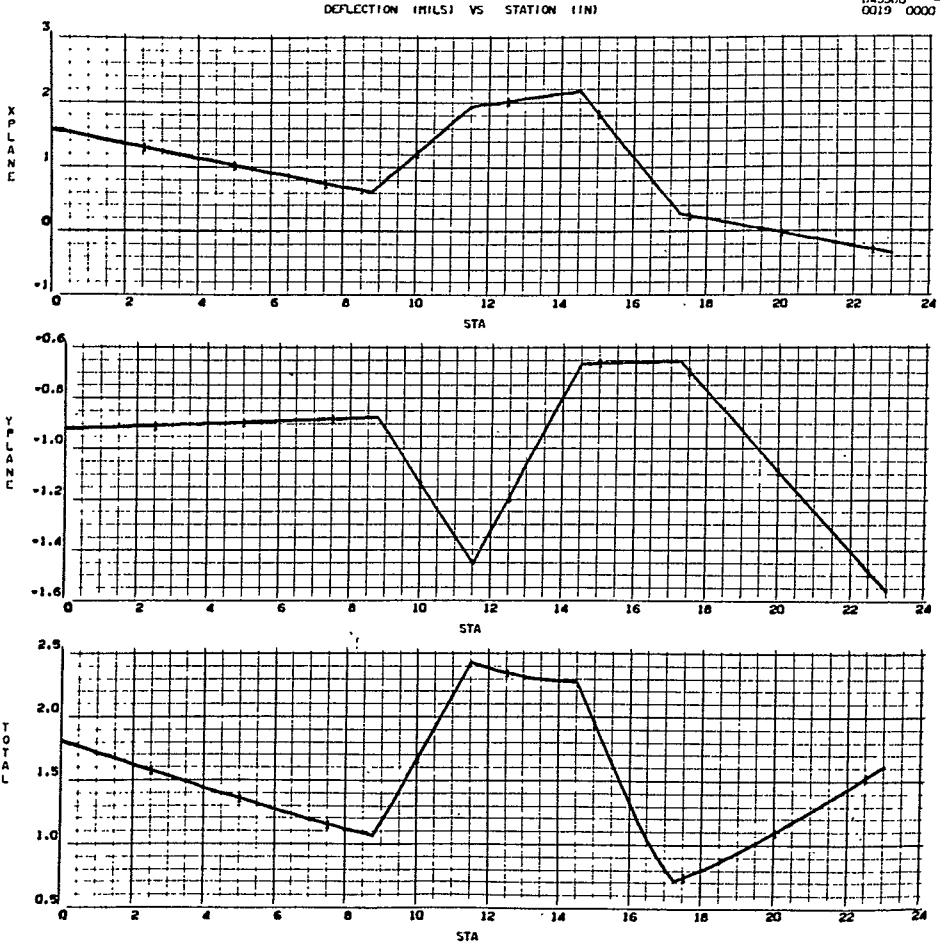


Figure 40

TEST 1144 30000 RPM

043980
0020 0000

DEFLECTION (MILS) VS STATION (IN)

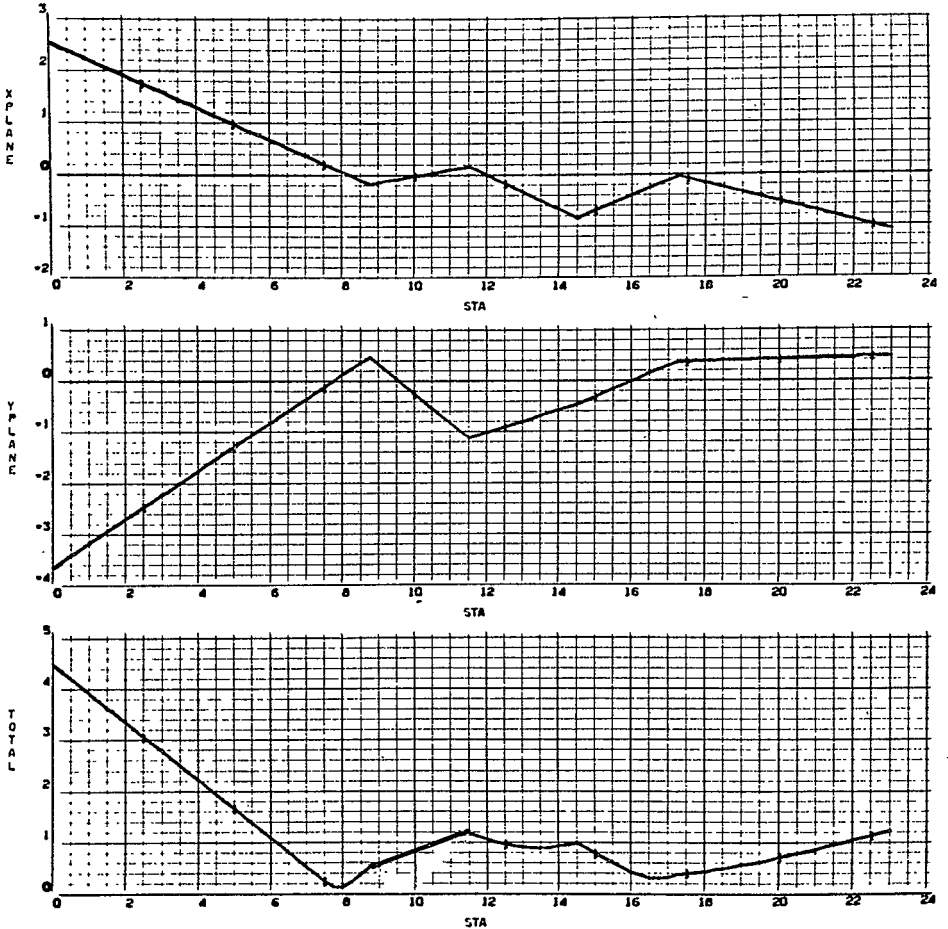
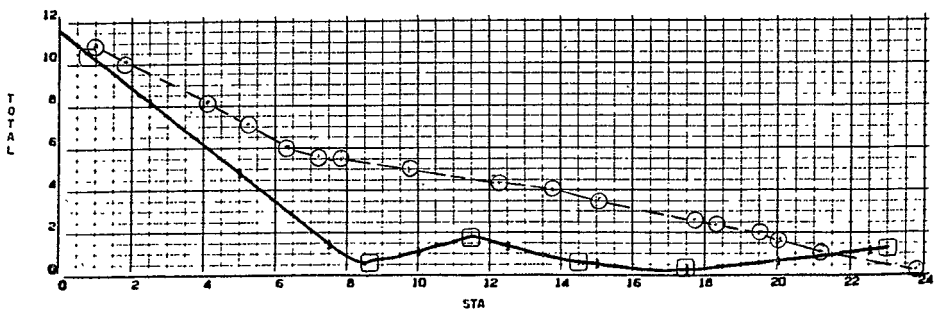
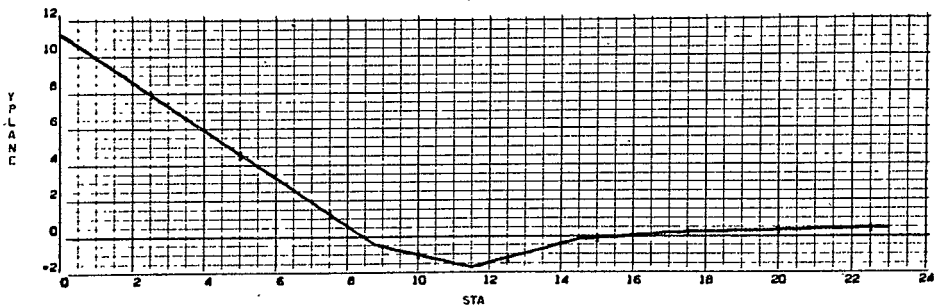
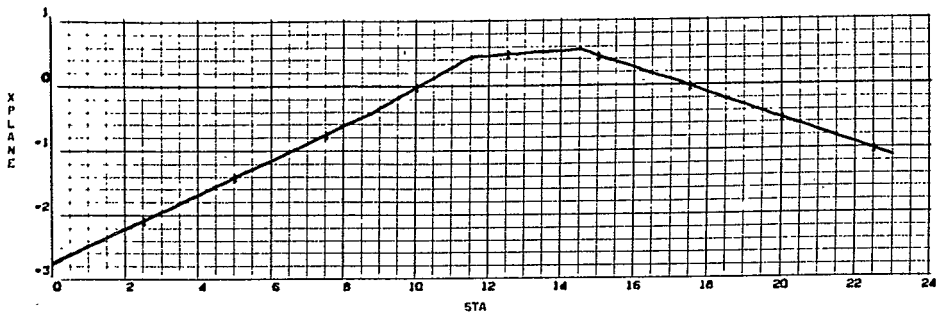


Figure 41

TEST 1145 26000 RPM

843980
0002 0000

DEFLECTION (MILS) VS STATION (IN)



○ COMPUTER SIMULATION DATA (x 40)

□ EXPERIMENTAL DATA

Figure 42.

3

TEST 1145

28000 RPM

843980
0001 0000

DEFLECTION (MILS) VS STATION (IN)

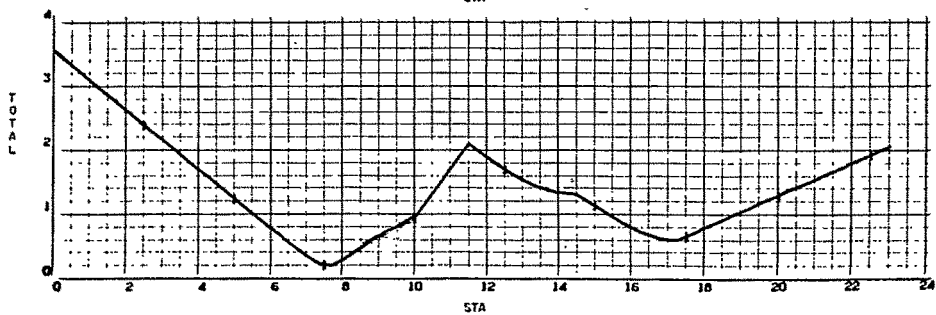
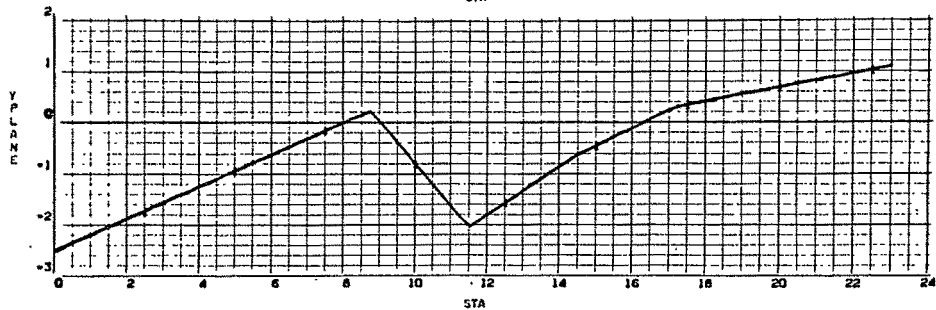
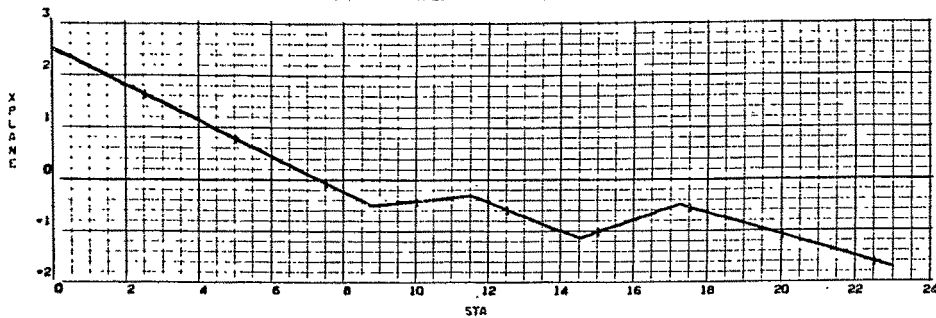


Figure 43

TEST 1145 30000 RPM

843980
0021 0000

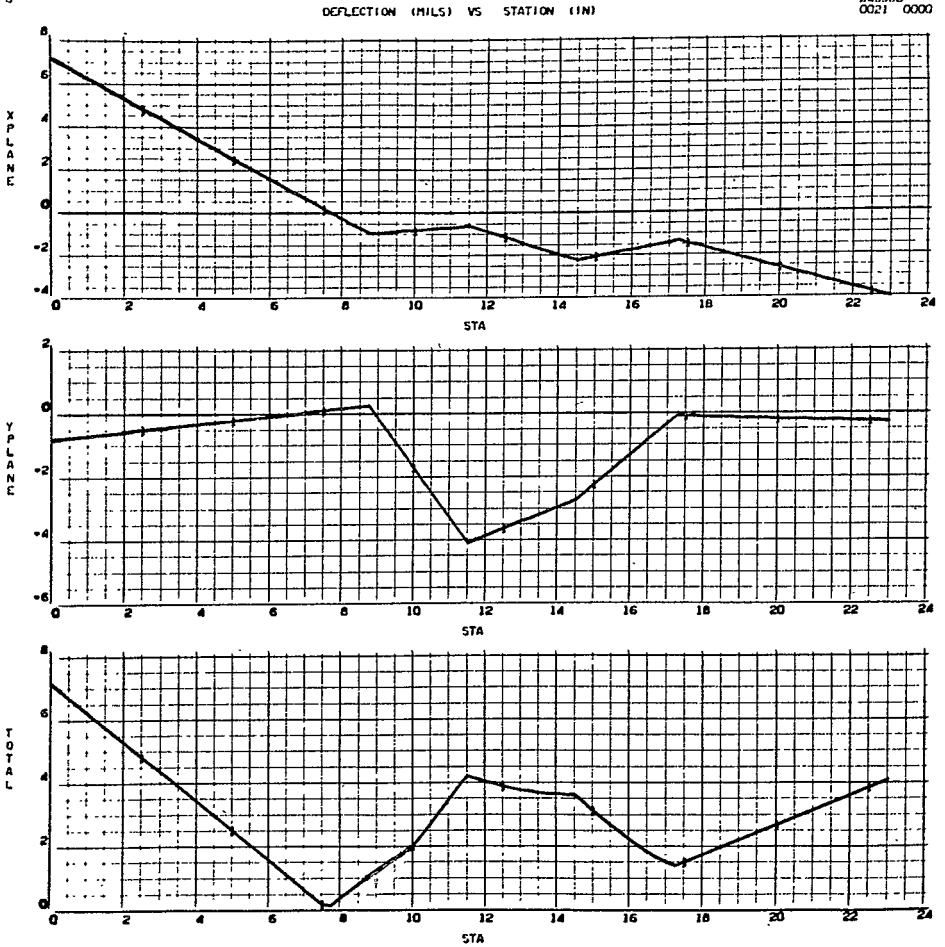


Figure 44

SIMULATION OF EXPERIMENTAL TESTS

SIMULATION MODEL

A discrete 17 mass model was constructed to describe the physical configuration of the Mark-25 pump rotor. Table 5 lists the mass and inertia properties for each of the 17 stations.

The four bearings, in two duplex pairs, were located at Stations 5, 6, 15 and 16. A speed sensitive linear stiffness characteristic was used for each of the four angular contact ball bearings.

Eight different cases of rotor mass unbalances corresponding to the experimental test runs were used in the computer program input data. These unbalance cases are listed in Table 6. Six steady state speed slices were simulated for each test run. The speeds were from 24,000 to 34,000 rpm in 2000 rpm increments.

RESULTS OF THE COMPUTER SIMULATION

The results of the computer simulation are listed in Tables 7 through 14. Although excellent correlation was achieved between the results of the new computer program versus a synchronous response program (see Table 2), large variances occurred in the simulation runs as compared to the experimental data.

For graphical comparison the simulation data is plotted on the experimental plots of deflection versus speed, and is identified by a small circle.* Also, comparison of the calculated vs. experimental mode shapes are shown in Figures 21, 24, 29, 32, 35 and 42. In some cases it was necessary to multiply the simulation data by a scale factor. However, the mode shapes compared favorably. The close agreement between the computer results and that from an existing synchronous rotordynamic response program suggests

*See Figures 13 through 20.

that the computer program is valid and able to generate accurate results. The discrepancies between the test and computer data should only result from inadequacies in test model simulation. Comparisons between the computer and test data (Figures 13 through 20) reveal,

1. Computer simulated deflections are generally smaller than the test data.
2. Within the speed range tested, reasonably negative curvatures (inverted U shape) of the deflection versus speed curves at the local maximum deflection regions are observed.
3. The local maximum deflection points from the computed data are in general at a higher speed than that from the test data.

Item 1 above suggests that the actual mass eccentricities of the test rotor may be greater than those used in the simulation. Since there are only a maximum of two unbalance locations for each unbalance configuration, it is possible that there are compensating local unbalances not detectable during the balance procedure. That is, the actual unbalance distribution of a rotor unbalance configuration is different from that used in the computer program. Although the postulate of actual eccentricities being larger than those used may tend to explain a good portion of the discrepancies between test and computer points, no such errors have been found in the test procedure.

Item 2 indicates that certain amount of damping exists. It appears that besides the small amount of inherent damping in the elastic bearing and mount components the high damping component Kirksite base for the test apparatus may contribute to the system as a significant damping parameter. These was no damping included in the computer analysis. The damping parameter has not been included in the STARUP subroutine.

Item 3 suggests that the support bearings stiffness and/or rotor rigidity are higher in the computer model than in the test mode, thus resulting in higher computed critical speeds. The test rotor is fabricated from individual disk components and is of reasonably complex structure. It is possible that the computer model may represent a more stiff rotor than the real one.

As far as the support bearing's stiffness simulation is concerned, the discrepancies between the actual value and that used in the computer program should not be very large. In the past experiences, the use of similar bearing stiffness appeared to give reasonable critical speed results comparable to those observed from related test data.

Tables 7 through 14 are copies of the actual printout from the computer simulation runs and list the following:

- a. X displacement array
- b. Y displacement array
- c. Rotor deflection vector array
- d. Rotor deflection vector phase angle array
- e. X velocity array
- f. Y velocity array
- g. Whirl frequency array
- h. Whirl to spin frequency ratio array

TABLE 5

MARK-25 PUMP ROTOR MASS AND INERTIA PROPERTIES

<u>Station</u>	<u>Location</u> <u>Inches</u>	<u>Mass</u> ₂ <u>lb-sec²/in</u>	<u>I_D</u> <u>lb-in-sec²</u>	<u>I_P</u> <u>lb-in-sec²</u>	<u>I_D - I_P</u>
1	1.0	.0023293	.0018504	.0022752	- .0004248
2	1.8	.008456	.027589	.047474	- .019885
3	4.02	.093548	.142081	.24449	- .10241
4	5.22	.017604	.052015	.089431	- .037416
5	6.32	.0056835	.0028311	.0046549	- .0018238
6	7.32	.0021558	.0010739	.0017656	- .0006917
7	7.81	.017136	.04451	.079042	- .049534
8	9.83	.054180	.1450	.2515	- .15662
9	12.24	.0331708	.0934	.1585	- .08504
10	13.73	.040357	.11800	.1914	- .09338
11	15.03	.0361565	.1019	.1834	- .10151
12	17.72	.030516	.0854	.1593	- .10791
13	18.32	.030516	.0854	.1593	- .10791
14	19.53	.021697	.0608	.1175	- .07673
15	20.02	.002661	.0037044	.0023834	+ .0013210
16	21.13	.010644	.014818	.009534	+ .005289
17	23.98	.011584	.012806	.013423	- .000617

TABLE 6

MARK-25 PUMP SIMULATION DATA

<u>Run</u>	<u>Station</u>	<u>Mass₂</u> <u>lb-sec²/in</u>	<u>Eccentricity, e</u>	<u>Phase,</u> <u>Degree</u>
1136	2	.008456	.00143"	138.0°
	5	.005684	.00341"	139.0°
1137	2	.008456	.00185"	195.0°
	5	.005684	.00425"	196.0°
1138	2	.008456	.000175"	-24.4°
	5	.005684	.000366"	-23.5°
1139	2	.008456	.000429"	-32.3°
	5	.005684	.0011"	-29.7°
1141	Base	Run	0.0"	0.0°
1142	10	.04036	.0002065"	121.0°
1143	17	.01158	.000058"	165.0°
1144	13	.03052	.0000964"	125.0°
1145	7	.01714	.000174"	115.0°

TABLE 7

TEST 1136 SIMULATION AT 26,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
	-3.3700D-03	-3.1209D-03	-2.4481D-03	-2.0794D-03	-1.7235D-03	-1.5516D-03
	-1.5324D-03	-1.3899D-03	-1.1926D-03	-1.0622D-03	-9.4029D-04	-6.7142D-04
	-6.10C5D-04	-4.8231D-04	-4.2161D-04	-2.9067D-04	2.9491D-05	
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
	2.8554D-03	2.6442D-03	2.0736D-03	1.7609D-03	1.4591D-03	1.3134D-03
	1.2971D-03	1.1763D-03	1.0092D-03	8.9876D-04	7.9555D-04	5.6788D-04
	5.1591D-04	4.0776D-04	3.5637D-04	2.4552D-04	-2.5563D-05	
ROTOR DEFLECTION VECTOR ARRAY, (IN.)						
	4.4170D-03	4.0904D-03	3.2083D-03	2.7248D-03	2.2582D-03	2.0329D-03
	2.0077D-03	1.8209D-03	1.5623D-03	1.3914D-03	1.2317D-03	8.7937D-04
	7.9895D-04	6.3158D-04	5.5204D-04	3.8049D-04	3.9028D-05	
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
111	1.3972D 02	1.3973D 02	1.3973D 02	1.3974D 02	1.3975D 02	1.3975D 02
	1.3975D 02	1.3976D 02	1.3976D 02	1.3976D 02	1.3977D 02	1.3978D 02
	1.3978D 02	1.3979D 02	1.3979D 02	1.3981D 02	3.1908D 02	
X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
	-7.7745D 00	-7.1994D 00	-5.6458D 00	-4.7944D 00	-3.9726D 00	-3.5758D 00
	-3.5317D 00	-3.2029D 00	-2.7478D 00	-2.4471D 00	-2.1661D 00	-1.5462D 00
	-1.4047D 00	-1.1102D 00	-9.7030D-01	-6.6846D-01	6.9622D-02	
Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
	-9.1755D 00	-8.4973D 00	-6.6655D 00	-5.6616D 00	-4.6927D 00	-4.2249D 00
	-4.1722D 00	-3.7842D 00	-3.2470D 00	-2.8920D 00	-2.5601D 00	-1.8281D 00
	-1.6610D 00	-1.3132D 00	-1.1479D 00	-7.9143D-01	8.0276D-02	
WHIRL FREQUENCY ARRAY, RPM						
	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04
	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04
	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04
WHIRL TO SPIN FREQUENCY RATIO ARRAY						
	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00
	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00
	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	9.9999D-01	

TABLE 8

TEST 1137 SIMULATION AT 26,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES.(IN.)						
	-5.3679D-03	-4.9705D-03	-3.8972D-03	-3.3091D-03	-2.7415D-03	-2.4676D-03
	-2.4369D-03	-2.2098D-03	-1.8957D-03	-1.6881D-03	-1.4942D-03	-1.0663D-03
	-9.6869D-04	-7.6546D-04	-6.6892D-04	-4.6066D-04	4.8728D-05	
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES.(IN.)						
	-1.6599D-03	-1.5373D-03	-1.2059D-03	-1.0243D-03	-8.4901D-04	-7.6437D-04
	-7.5490D-04	-6.8470D-04	-5.8751D-04	-5.2327D-04	-4.6324D-04	-3.3079D-04
	-3.0056D-04	-2.3764D-04	-2.0773D-04	-1.4323D-04	1.4487D-05	
ROTOR DEFLECTION VECTOR ARRAY (IN.)						
	5.6187D-03	5.2028D-03	4.0795D-03	3.4640D-03	2.8700D-03	2.5833D-03
	2.5511D-03	2.3135D-03	1.9846D-03	1.7674D-03	1.5643D-03	1.1165D-03
	1.0142D-03	8.0150D-04	7.0043D-04	4.8241D-04	5.0835D-05	
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
611	1.9718D 02	1.9719D 02	1.9719D 02	1.9720D 02	1.9721D 02	1.9721D 02
	1.9721D 02	1.9722D 02	1.9722D 02	1.9722D 02	1.9723D 02	1.9723D 02
	1.9724D 02	1.9725D 02	1.9725D 02	1.9727D 02	1.6557D 01	
X VELOCITY ARRAY IN STATIONARY COORDINATES,(IN/SEC)						
	4.5196D 00	4.1856D 00	3.2834D 00	2.7889D 00	2.3120D 00	2.0817D 00
	2.0554D 00	1.8643D 00	1.5996D 00	1.4247D 00	1.2613D 00	9.0065D-01
	8.1833D-01	6.4702D-01	5.6558D-01	3.9004D-01	-3.9393D-02	
Y VELOCITY ARRAY IN STATIONARY COORDINATES,(IN./SEC)						
	-1.4615D 01	-1.3533D 01	-1.0611D 01	-9.0097D 00	-7.4643D 00	-6.7184D 00
	-6.6349D 00	-6.0168D 00	-5.1614D 00	-4.5963D 00	-4.0682D 00	-2.9033D 00
	-2.6375D 00	-2.0841D 00	-1.8213D 00	-1.2542D 00	1.3268D-01	
WHIRL FREQUENCY ARRAY,RPM						
	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04
	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04
	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	
WHIRL TC SPIN FREQUENCY RATIO ARRAY						
	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00
	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00
	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	9.9999D-01	

TABLE 9

TEST 1138 SIMULATION AT 32,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
-8.9574D-04	-7.9473D-04	-5.2262D-04	-3.7265D-04	-2.2741D-04	-1.2549D-04	
-3.7334D-05	4.1334D-05	1.8349D-04	2.6620D-04	3.3372D-04	4.6182D-04	
4.8891D-04	5.3972D-04	5.5193D-04	5.9336D-04	7.6412D-04		
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
3.8902D-04	3.2740D-04	2.1530D-04	1.5352D-04	9.3674D-05	5.1662D-05	
3.5924D-05	-1.7135D-05	-7.5755D-05	-1.0986D-04	-1.3770D-04	-1.9052D-04	
-2.0169D-04	-2.2264D-04	-2.2767D-04	-2.4475D-04	-3.1516D-04		
ROTOR DEFLECTION VECTOR ARRAY (IN.)						
9.8878D-04	8.5953D-04	5.6523D-04	4.0303D-04	2.4594D-04	1.3571D-04	
9.4434D-05	4.4745D-05	1.9851D-04	2.8798D-04	3.6102D-04	4.9958D-04	
5.2887D-04	5.8384D-04	5.9765D-04	6.4185D-04	8.2656D-04		
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
120	1.5761D 02	1.5761D 02	1.5761D 02	1.5761D 02	1.5761D 02	1.5762D 02
	3.3758D 02	3.3758D 02	3.3757D 02	3.3757D 02	3.3758D 02	3.3758D 02
	3.3758D 02	3.3758D 02	3.3758D 02	3.3758D 02	3.3759D 02	
X-VELOCITY ARRAY IN STATIONARY COORDINATES, (IN/SEC)						
-1.2366D 00	-1.0971D 00	-7.2142D-01	-5.1453D-01	-3.1323D-01	-1.7316D-01	
-1.2043D-01	5.7400D-02	2.5383D-01	3.6812D-01	4.6139D-01	6.3824D-01	
6.7558D-01	7.4629D-01	7.6659D-01	8.2080D-01	1.0569D 00		
Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN/SEC)						
-3.0017D 00	-2.6632D 00	-1.7513D 00	-1.2487D 00	-7.6234D-01	-4.2052D-01	
-2.9264D-01	1.3852D-01	6.1490D-01	8.9205D-01	1.1183D 00	1.5477D 00	
1.6385D 00	1.8085D 00	1.8506D 00	1.9881D 00	2.5602D 00		
WHIRL FREQUENCY ARRAY, RPM						
3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2001D 04	3.2000D 04	
3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	
3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04		
WHIRL TO SPIN FREQUENCY RATIO ARRAY						
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	9.9999D-01	
9.9999D-01	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	
1.0000D 00	9.9999D-01	1.0000D 00	9.9999D-01	9.9999D-01		

TABLE 10

TEST 1139 SIMULATION AT 32,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)

-2.1879D-03	-1.9412D-03	-1.2765D-03	-9.1032D-04	-5.5573D-04	-3.0770D-04
-2.1501D-04	9.7638D-05	4.4312D-04	6.4415D-04	8.0830D-04	1.1198D-03
1.1856D-03	1.3093D-03	1.3390D-03	1.4398D-03	1.8550D-03	

Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)

1.2669D-03	1.1241D-03	7.3920D-04	5.2711D-04	3.2173D-04	1.7785D-04
1.2403D-04	-5.7470D-05	-2.5802D-04	-3.7470D-04	-4.6998D-04	-6.5074D-04
-6.8896D-04	-7.6068D-04	-7.7794D-04	-8.3641D-04	-1.0774D-03	

ROTOR DEFLECTION VECTOR ARRAY (IN.)

2.5282D-03	2.2431D-03	1.4751D-03	1.0519D-03	6.4214D-04	3.5540D-04
2.4622D-04	1.1330D-04	5.1277D-04	7.4520D-04	9.3500D-04	1.2951D-03
1.3713D-03	1.5142D-03	1.5486D-03	1.6651D-03	2.1451D-03	

PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES

121	1.4993D 02	1.4993D 02	1.4993D 02	1.4993D 02	1.4993D 02	1.4997D 02
	1.5002D 02	3.2952D 02	3.2979D 02	3.2981D 02	3.2982D 02	3.2984D 02
	3.2984D 02	3.2984D 02	3.2984D 02	3.2985D 02	3.2985D 02	

X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN/SEC)

-4.2455D 00	-3.7667D 00	-2.4770D 00	-1.7666D 00	-1.0768D 00	-5.9614D-01
-4.1573D-01	1.9254D-01	8.6457D-01	1.2556D 00	1.5748D 00	2.1803D 00
2.3081D 00	2.5498D 00	2.6017D 00	2.8042D 00	3.6118D 00	

Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN/SEC)

-7.3317D 00	-6.5050D 00	-4.2778D 00	-3.0504D 00	-1.8631D 00	-1.0310D 00
-7.2045D-01	3.2721D-01	1.4850D 00	2.1586D 00	2.7087D 00	3.7527D 00
3.9736D 00	4.3869D 00	4.4902D 00	4.8240D 00	6.2150D 00	

WHIRL FREQUENCY ARRAY, RPM

3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2001D 04	3.2000D 04
3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04
3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	

WHIRL TO SPIN FREQUENCY RATIO ARRAY

1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	9.9999D-01
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00
1.0000D 00	9.9999D-01	1.0000D 00	9.9999D-01	9.9999D-01	

TABLE 11

TEST 1142 SIMULATION AT 52,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
-7.5056D-04	-8.6020D-04	-4.16C1D-04	-2.8211D-04	-1.5348D-04	-6.0035D-05	
-2.3716D-05	9.5442D-05	2.2516D-04	2.9994D-04	3.6124D-04	4.7614D-04	
5.0023D-04	5.4497D-04	5.5465D-04	5.9021D-04	7.4478D-04		
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
1.1668D-03	1.0263D-03	6.4671D-04	4.3856D-04	2.3859D-04	9.3327D-05	
3.6866D-05	-1.4837D-04	-3.5002D-04	-4.6628D-04	-5.6157D-04	-7.4018D-04	
-7.7764D-04	-8.4719D-04	-8.6223D-04	-9.1751D-04	-1.1578D-03		
ROTOR DEFLECTION VECTOR ARRAY (IN.)						
1.3873D-03	1.2203D-03	7.6856D-04	5.2146D-04	2.8370D-04	1.1097D-04	
4.3837D-05	1.7642D-04	4.1619D-04	5.5442D-04	6.6772D-04	8.8010D-04	
9.2464D-04	1.0073D-03	1.0252D-03	1.0910D-03	1.3767D-03		
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
122	1.2275D 02	1.2275D C2	1.2275D 02	1.2275D C2	1.2275D 02	1.2275D 02
	1.2275D 02	3.0275D C2	3.0275D 02	3.0275D 02	3.0275D 02	3.0275D 02
	3.0275D 02	3.0275D C2	3.0275D C2	3.0275D 02	3.0275D 02	
X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-3.9099D 00	-3.4392D C0	-2.1671D 00	-1.4697D 00	-7.9891D-01	-3.1278D-01	
-1.2359D-01	4.9718D-01	1.1729D 00	1.5625D 00	1.8818D 00	2.4802D 00	
2.6056D 00	2.8391D C0	2.8871D 00	3.0752D C0	3.8806D 00		
Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-2.5152D 00	-2.2124D C0	-1.3941D 00	-9.4523D-01	-5.1536D-01	-2.0110D-01	
-7.9393D-02	3.1986D-01	7.5456D-01	1.0052D 00	1.2106D C0	1.5959D 00	
1.6767D 00	1.8259D C0	1.8623D 00	1.9768C 00	2.4945D 00		
WHIRL FREQUENCY ARRAY, RPM						
3.2000D 04	3.2000D C4	3.2000D 04	3.2000D 04	3.2001D 04	3.1999D 04	
3.1999D 04	3.2000D C4	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	
3.2000D 04	3.2000D C4	3.2000D 04	3.2000D 04	3.2000D 04		
WHIRL TO SPIN FREQUENCY RATIO ARRAY						
1.0000D 00	1.0000D C0	1.0000D 00	1.0000D 00	1.0000D 00	9.9998D-01	
9.9998D-01	1.0000D C0	1.0000D C0	1.0000D 00	1.0000D 00	1.0000D 00	
1.0000D 00	9.9999D-01	1.0000D 00	9.9999D-01	9.9999D-01		

TABLE 12

TEST 1143 SIMULATION AT 32,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)

-3.0738D-04	-2.7112D-04	-1.7318D-04	-1.1944D-04	-6.7755D-05	-3.0700D-05
-1.6576D-05	2.9509D-05	7.9637D-05	1.0850D-04	1.3181D-04	1.7536D-04
1.8448D-04	2.0137D-04	2.0486D-04	2.1706D-04	2.6845D-04	

Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)

7.2367D-05	6.3830D-05	4.0771D-05	2.8121D-05	1.5952D-05	7.2277D-06
3.9025D-06	-6.9473D-06	-1.8749D-05	-2.5545D-05	-3.1031D-05	-4.1284D-05
-4.3433D-05	-4.7408D-05	-4.8231D-05	-5.1102D-05	-6.3201D-05	

ROTOR DEFLECTION VECTOR ARRAY (IN.)

3.1578D-04	2.7853D-04	1.7791D-04	1.2271D-04	6.9608D-05	3.1539D-05
1.7029D-05	3.0316D-05	8.1814D-05	1.1147D-04	1.3541D-04	1.8015D-04
1.8953D-04	2.0687D-04	2.1046D-04	2.2299D-04	2.7578D-04	

PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES

1.6675D 02	1.6675D 02	1.6675D 02	1.6675D 02	1.6675D 02	1.6675D 02
1.6675D 02	3.4675D 02	3.4675D 02	3.4675D 02	3.4675D 02	3.4675D 02
3.4675D 02	3.4675D 02	3.4675D 02	3.4675D 02	3.4675D 02	3.4675D 02

X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)

-2.4249D-01	-2.1388D-01	-1.3660D-01	-9.4266D-02	-5.3219D-02	-2.4235D-02
-1.3096D-02	2.3274D-02	6.2819D-02	8.5592D-02	1.0397D-01	1.3827D-01
1.4545D-01	1.5895D-01	1.6080D-01	1.7147D-01	2.1208D-01	

Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)

-1.0300D 00	-9.0853D-01	-5.8032D-01	-4.0025D-01	-2.2711D-01	-1.0287D-01
-5.5542D-02	9.8887D-02	2.6687D-01	3.6360D-01	4.4169D-01	5.8764D-01
6.1824D-01	6.7476D-01	6.8671D-01	7.2730D-01	8.9949D-01	

WHIRL FREQUENCY ARRAY, RPM

3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2001D 04	3.2000D 04
3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04
3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	

WHIRL TO SPIN FREQUENCY RATIO ARRAY

1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	9.9999D-01
9.9999D-01	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00
1.0000D 00	9.9999D-01	1.0000D 00	9.9999D-01	9.9999D-01	

TABLE 13

TEST 1144 SIMULATION AT 32,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
-5.50820-04	-4.67780-04	-2.97490-04	-2.04080-04	-1.14270-04	-4.95970-05	
-2.47870-05	5.63150-05	1.44560-04	1.95390-04	2.36440-04	3.13270-04	
5.29430-04	3.59700-04	3.66390-04	3.90570-04	4.94610-04		
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
7.10000-04	6.26390-04	3.98360-04	2.73280-04	1.53010-04	6.64140-05	
3.51920-05	-7.65090-05	-1.92570-04	-2.61640-04	-3.16600-04	-4.19490-04	
-4.64130-04	-4.81670-04	-4.90620-04	-5.23000-04	-6.62310-04		
ROTOR DEFLECTION VECTOR ARRAY (IN.)						
8.87150-04	7.81780-04	4.97180-04	3.41080-04	1.90970-04	8.28890-05	
4.14260-05	9.41160-05	2.41590-04	3.26540-04	3.95140-04	5.23560-04	
5.50570-04	6.01160-04	6.12230-04	6.52740-04	8.26610-04		
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
121	1.26750 02	1.26750 02	1.26750 02	1.26750 02	1.26750 02	1.26750 02
	1.26750 02	3.06750 02	3.06750 02	3.06750 02	3.06750 02	3.06750 02
	3.06750 02	3.06750 02	3.06750 02	3.06750 02	3.06750 02	3.06750 02
X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-2.98150 00	-2.09900 00	-1.33490 00	-9.15830-01	-5.12330-01	-2.22580-01	
-1.11260-01	2.52690-01	6.48640-01	8.76730-01	1.06090 00	1.40560 00	
1.47810 00	1.61420 00	1.64260 00	1.75300 00	2.21990 00		
Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-1.77880 00	-1.56760 00	-9.96550-01	-6.83810-01	-3.83510-01	-1.66160-01	
-8.36170-02	1.88730-01	4.84440-01	6.54770-01	7.92350-01	1.05000 00	
1.10420 00	1.20520 00	1.22990 00	1.30820 00	1.65670 00		
WHIRL FREQUENCY ARRAY, RPM						
3.20000 04	3.20000 04	3.20000 04	3.20000 04	3.20010 04	3.20000 04	
3.20000 04	3.20000 04	3.20000 04	3.20000 04	3.20000 04	3.20000 04	
3.20000 04	3.20000 04	3.20010 04	3.20000 04	3.20000 04		
WHIRL TO SPIN FREQUENCY RATIO ARRAY						
1.00000 00	1.00000 00	1.00000 00	1.00000 00	1.00000 00	9.99990-01	
9.99990-01	1.00000 00	1.00000 00	1.00000 00	1.00000 00	1.00000 00	
1.00000 00	9.99990-01	1.00000 00	9.99990-01	9.99990-01		

TABLE 14

TABLE 14.5 SIMULATION AT 26,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
-1.2147D-04	-1.1321D-04	-9.1C15D-05	-7.8748D-05	-6.6757D-05	-6.1523D-05	
-6.1292D-05	-5.6349D-05	-4.9176D-05	-4.4338D-05	-3.9749D-05	-2.9490D-05	
-2.7140D-05	-2.2215D-05	-1.9755D-05	-1.4632D-05	-2.3146D-06		
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
2.4444D-04	2.2783D-04	1.8316D-04	1.5847D-04	1.3434D-04	1.2381D-04	
1.2335D-04	1.1340D-04	9.8563D-05	8.9226D-05	7.9992D-05	5.9347D-05	
5.4617D-05	4.4706D-05	3.9835D-05	2.9447D-05	4.6579D-06		
ROTOR DEFLECTION VECTOR ARRAY (IN.)						
2.7296D-04	2.5440D-04	2.0453D-04	1.7696D-04	1.5002D-04	1.3825D-04	
1.3775D-04	1.2663D-04	1.1C51D-04	9.9634D-05	8.9324D-05	6.6270D-05	
6.0988D-05	4.9921D-05	4.4482D-05	3.2882D-05	5.2013D-06		
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
1.1642D 02	1.1642D 02	1.1642D 02	1.1642D 02	1.1642D 02	1.1642D 02	
1.1642D 02	1.1642D 02	1.1642D 02	1.1642D 02	1.1642D 02	1.1642D 02	
1.1642D 02	1.1642D 02	1.1642D 02	1.1642D 02	1.1642D 02	1.1642D 02	
X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-6.6555D-01	-6.2030D-01	-4.9869D-01	-4.3148D-01	-3.6577D-01	-3.3709D-01	
-3.3583D-01	-3.0875D-01	-2.6945D-01	-2.4294D-01	-2.1779D-01	-1.6158D-01	
-1.4670D-01	-1.2172D-01	-1.0845D-01	-8.0175D-02	-1.2684D-02		
Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-3.3072D-01	-3.0824D-01	-2.4781D-01	-2.1441D-01	-1.8177D-01	-1.6754D-01	
-1.6688D-01	-1.5342D-01	-1.3389D-01	-1.2072D-01	-1.0823D-01	-8.0295D-02	
-7.5896D-02	-6.0484D-02	-5.3910D-02	-3.9839D-02	-6.2989D-03		
WHIRL FREQUENCY ARRAY, RPM						
2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	
2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	
2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	
WHIRL TO SPIN FREQUENCY RATIO ARRAY						
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	9.9999D-01		

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CONCLUSIONS AND RECOMMENDATIONS

CONCLUSIONS

The Rotordynamics Computer program, as developed under this contract, can serve as a useful tool in predicting the dynamic performance of a rotor-bearing system under various operating conditions. The drive and dissipation torque effects on an accelerating or decelerating rotor can be analyzed with this program.

In addition, the causes and effects of non-synchronous whirl rotor motion may also be studied by applying appropriate out-of-phase stiffness and positive or negative damping excitations. The computer program can be used as an analytical study tool for a general transient spin-speed and/or non-axisymmetric rotor motion.

The computer results of the simulation did not show good agreement, in general, with the test data. Some of the causes for the poor correlation may have been due to unknown characteristics not included in the simulation model. Some of these characteristics are:

1. unknown damping coefficient associated with bearing,
2. mount damping,
3. test mount pedestal stiffness and mass considerations,
4. bearing non-linear description and
5. internal hysteretic damping of the rotor.

In addition, the original intent to iterate the model was not funded; therefore, the time and funds to accomplish improved correlation between the computer results and the experimental data was not available. These items are discussed in the recommendations.

RECOMMENDATIONS

The period of the current contract provided adequate time to update the computer program and to obtain verification. An immediate follow-on effort is needed to study and correlate the integration process with the basic mathematical formulation such that a thorough understanding may be achieved. This would lead to possible improvement in computation-to-real time ratio. Other items needed to be included are the transverse dissipation, the out-of-phase stiffness and damping functions, and the non-linear stiffness bearing in the "STARUP" subroutine.

In addition, an iteration or parametric study is required to improve the Mark-25 model so that a better degree of correlation may be achieved between the simulation results and experimental data.

Recommendations of continued future efforts to improve capability of the computer program are divided into 3 groups, with group I representing the most important items, then group II, and then group III with the least important items.

Group I:

- a. In depth study of the integrations between the basic mathematical formulation and integration process in an attempt to improve computation-to-real time ratio.
- b. Up-date the "STARUP" subroutine to include other rotordynamics parameters such as:
 1. Rotor drive torque and dissipation functions.
 2. Rotor in-phase damping and in-phase and out-of-phase stiffness and damping parameters.

3. Non-linear bearing stiffness characteristics.

Group II:

- a. Inclusion of the Bearing Mass Parameter. The bearing mass will be simulated by a concentrated mass located between a bearing and its mount. For a bearing with soft mount design and particularly in high speed operation, the bearing mass effects should be considered. In addition to the bearing mass parameter, the bearing transverse mass moment of inertia also affects the rotordynamic performance. For normal small operating bearing misalignment, the effects of the transverse mass moment of inertia parameter are relatively small and its inclusion in the rotordynamic study would be optional.

- b. Formulation of Hydrodynamic Force Excitation Functions. Particular emphases are placed on the axial pump impeller to stator interactions. A linear hydrodynamic force characteristics should be sufficient at this stage of the program development. These force characteristics will include in-phase and out-of-phase excitations which may also be functions of spin and/or whirl frequencies. The hydrodynamic excitations of the close-clearance flow-control surfaces of the balance piston such as that used in the Mark-25 axial pumps were found to be small compared to other rotordynamic loads and hence they will not be included.

- c. Feasibility Study of Advanced Rotating-Coordinate Systems. A feasibility study is suggested to investigate the practicality of using advanced rotating coordinates to facilitate the integration process by smoothing out the second order rotordynamic function fluctuations. The second order function

fluctuations may result from cyclic force variations of the rotor mass unbalance in a non-synchronous whirl rotor motion or from non-isotropic bearing mount force characteristics.

Group III:

Inclusion of the below listed parameters as discussed:

- a. Rotor Casing Mass and Transverse Mass Moment of Inertia, and the Casing Support Stiffness and Damping Parameter. In these suggested parameters, a rigid casing is assumed. The casing is considered to be supported on an inertial frame of reference (foundation) through elastic and damping members. The stiffness and damping parameters include both force and moment functions of displacements and misalignments respectively.
- b. Rotor Material Hysteresis. This parameter includes also the effects of press-fit and/or bolted joints in an elastic rotor. This hysteresis parameter will include both in-phase and out-of-phase damping characteristics.
- c. Transverse Sinusoidal Ambient Vibrations. A fixed-amplitude and constant-frequency sinusoidal vibration is considered to be imposed on an otherwise fixed foundation. It appears that for a rotor-bearing system operating under a high-frequency ambient vibration condition, the computer time may become excessive unless only the steady-state rotordynamic performance is studied. An approximate treatment in including the ambient vibration parameter is to apply an equivalent cyclic G-loading directly to the rotor. This treatment would substantially reduce the complexity of the formulation required for the inclusion of the ambient vibration parameter.

APPENDIX A

A 5-STATION 5-MASS TEST ROTOR MODEL, BEARINGS LOCATED AT STATIONS 1 AND
5
LOS ANGELES, CALIF. APRIL 4/1970

NS = 5 NUMBER OF ROTOR STATIONS (ALLOWABLE RANGE: 5=<NS=<25)
IASIGN = 1 A ROTOR STATION NUMBER AT WHICH THE WHIRL/SPIN FREQUENCY RATIO WILL BE
PLOTTED ON CRT
NOORPM = 1 THE NUMBER OF SPIN SPEEDS IN RPM AT OR NEAR WHICH 3-DIMENSIONAL ABSOLUTE ROTOR
MODE SHAPE CRT GRAPHS ARE REQUIRED. THE SPIN SPEED RPM VALUES ARE LISTED
UNDER INPRPM ARRAY.
(ALLOWABLE RANGE: 0=<NOORPM=<50)
NPOINT = 25 THE NUMBER OF POINTS (ONE PFP EACH INTEGRATION STEP) FOR EACH CRT GRAPH.
(ALLOWABLE RANGE: 1=<NPOINT=<50)
CRT = 0 CRT=0 MEANS CRT IS NOT REQUIRED
CRT=1 MEANS CRT IS REQUIRED
ACCEL = 0 ACCEL=0 MEANS A 3-DIMENSIONAL ROTOR MODE SHAPE CRT CORRESPONDING TO THAT AT BEG-
INNING OF THE RUN WOULD BE PROVIDED IF CONCURRENTLY CRT=1.
ACCEL=1 MEANS ONLY THE TRANSIENT-SPEED ROTOR MODE SHAPES AT OR NEAR INPRPM
VALUES WILL BE PROVIDED IF CONCURRENTLY CRT = 1.
INPRPM ARRAY THE ROTOR SPIN SPEED RPM VALUES AT OR NEAR WHICH CRT GRAPHS FOR
3-DIMENSIONAL ABSOLUTE ROTOR MODE SHAPES ARE REQUIRED.
0.0
T = 0.0 INITIAL REAL TIME, SEC.
DT = 1.0000D-05 ESTIMATED INITIAL INTEGRATION STEP (REAL) TIME, SEC.
TMAX = 2.0000D-02 TOTAL REAL TIME TO BE RUN, SEC.
TOLI = 1.0000D-02 INTEGRATION TOLERANCE, FRACTION
TOLR = 1.0000D-03 TOLERANCE IN COMPUTING BEARING DISPLACEMENTS, FRACTION
TSTDP = 5.0000D-01 THE COMPUTER TIME ALLOWED FOR EACH SET OF DATA, MINUTES.
GX = 0.0 GRAVITY OR G-LOADING IN X DIRECTION, IN/SEC**2
GY = 0.0 GRAVITY OR G-LOADING IN Y DIRECTION, IN/SEC**2
TM7 = 0.0 EXPONENT FOR SPEED SENSITIVE ROTOR DRIVE TORQUE
TMZ1 = 0.0 COEFFICIENT FOR ROTOR DRIVE TORQUE
TMZ2 = 0.0 COEFFICIENT FOR ROTOR DRIVE TORQUE, (IN-LB-SEC)/RAD.
TMZ3 = 0.0 COEFFICIENT FOR ROTOR DRIVE TORQUE, IN-LB

DD ARRAY OUTSIDE DIAMETERS OF ROTOR SECTIONS BETWEEN ADJACENT ROTOR STATIONS, IN.
1.00000 01 1.00000 01 1.00000 01 1.00000 01

D ARRAY INSIDE DIAMETERS OF ROTOR SECTIONS BETWEEN ADJACENT ROTOR STATIONS, IN.
0.0 0.0 0.0 0.0

QL ARRAY ROTOR SECTION LENGTHS BETWEEN ADJACENT ROTOR STATIONS, IN.
1.00000 00 1.00000 00 1.00000 00 1.00000 00

DN ARRAY MATERIAL DENSITIES OF ROTOR SECTIONS BETWEEN ADJACENT STATIONS, LB/IN**3
3.00000-01 3.00000-01 3.00000-01 3.00000-01

EE ARRAY YOUNGS MODULI OF ROTOR SECTIONS BETWEEN ADJACENT STATIONS, LB/IN**2
0.0 0.0 0.0 0.0

GG ARRAY SHEAR MODULI OF ROTOR SECTIONS BETWEEN ADJACENT STATIONS, LB/IN**2
1.15000 07 1.15000 07 1.15000 07 1.15000 07

131 EI ARRAY DIRECT INPUT OF THE PRODUCTS OF YOUNGS MODULI AND AREA MOMENTS OF INERTIA OF ROTOR
 SECTIONS BETWEEN ADJACENT STATIONS, LB-IN**2
1.66670 04 1.66670 04 1.66670 04 1.66670 04

GAK ARRAY DIRECT INPUT OF THE PRODUCTS OF SHEAR MODULI, CROSS-SECTIONAL AREAS
 AND RECIPROCAL OF SHEAR STRESS CONCENTRATION FACTORS BETWEEN ADJACENT STATIONS, LB.
0.0 0.0 0.0 0.0

AM ARRAY ADDITIONAL ROTOR MASSES AT ROTOR STATIONS, (LB-SEC**2)/IN.
0.0 0.0 0.0 0.0 0.0

AID ARRAY ADDITIONAL ROTOR TRANSVERSE MASS MOMENTS OF INERTIA AT ROTOR STATIONS, (LB-IN-SEC**2)
0.0 0.0 0.0 0.0 0.0

AIRD ARRAY ADDITIONAL ROTOR POLAR MASS MOMENTS OF INERTIA AT ROTOR STATIONS, (LB-IN-SEC**2)
1.00000-16 1.00000-16 1.00000-16 1.00000-16 1.00000-16

FCC ARRAY ROTOR MASS ECCENTRICITIES AT ROTOR STATIONS, IN.
1.00000-03 1.00000-03 1.00000-03 1.00000-03 1.00000-03

ALFA ARRAY PHASE ANGLES FOR ROTOR MASS ECCENTRICITY VECTORS AT ROTOR STATIONS MEASURED FROM THE
 INITIAL ROTOR SPIN ANGULAR POSITION, DEGREES
4.50000 01 4.50000 01 4.50000 01 4.50000 01

BETA ARRAY	INITIAL MISALIGNMENTS BETWEEN THE AXES OF THE MASS MOMENTS OF INERTIA AND THE ELASTIC AXES AT ROTOR STATIONS, DEGREES
0.0	0.0 0.0 0.0 0.0
GAMMA ARRAY	ANGULAR POSITIONS OF THE X-Y PLANE PROJECTIONS OF THE AXES OF MASS MOMENTS OF INERTIA AT ROTOR STATIONS MEASURED FROM THAT AT THE FIRST ROTOR STATION, DEGREES
0.0	0.0 0.0 0.0 0.0
C7 ARRAY	TORSIONAL FRICTION EXPONENTS AT ROTOR STATIONS, DIMENSIONLESS
0.0	0.0 0.0 0.0 0.0
CZ1 ARRAY	TORSIONAL FRICTION COEFFICIENTS AT ROTOR STATIONS, DIMENSION OF CZ1(I)*FDOT**CZ(I) IS IN-LB.
0.0	0.0 0.0 0.0 0.0
C72 ARRAY	TORSIONAL_FRICTION COEFFICIENTS AT ROTOR STATIONS, (IN-LB-SEC)/RAD.
0.0	0.0 0.0 0.0 0.0
XKF ARRAY	WHIRL-FREQUENCY FACTORS FOR STIFFNESS FORCE COEFFICIENTS AT ROTOR STATIONS, DIMENSIONLESS
0.0	0.0 0.0 0.0 0.0
XCF ARRAY	WHIRL-FREQUENCY FACTORS FOR DAMPING FORCE COEFFICIENTS AT ROTOR STATIONS, DIMENSIONLESS
0.0	0.0 0.0 0.0 0.0
XXFF ARRAY	WHIRL-FREQUENCY FACTORS FOR STIFFNESS MOMENT COEFFICIENTS AT ROTOR STATIONS, DIMENSIONLESS
0.0	0.0 0.0 0.0 0.0
XCFF ARRAY	WHIRL-FREQUENCY FACTORS FOR DAMPING MOMENT COEFFICIENTS AT ROTOR STATIONS, DIMENSIONLESS
0.0	0.0 0.0 0.0 0.0
QK ARRAY	IN-PHASE STIFFNESS FORCE COEFFICIENTS AT ROTOR STATIONS, LB/IN.
0.0	0.0 0.0 0.0 0.0
QC ARRAY	IN-PHASE DAMPING FORCE COEFFICIENTS AT ROTOR STATIONS, (LB-SEC)/IN.
0.0	0.0 0.0 0.0 0.0

QKP ARRAY	OUT-OF-PHASE STIFFNESS FORCE COEFFICIENTS AT ROTOR STATIONS, LB/IN.
0.0	0.0 0.0 0.0 0.0
QCP ARRAY	OUT-OF-PHASE DAMPING FORCE COEFFICIENTS AT ROTOR STATIONS, (LB-SEC)/IN.
0.0	0.0 0.0 0.0 0.0
Q<HD ARRAY	OUT-OF-PHASE WHIRL AND SPIN VELOCITY SENSITIVE STIFFNESS FORCE COEFFICIENTS AT ROTOR STATIONS, (LB-SEC)/(IN-RAD)
0.0	0.0 0.0 0.0 0.0
QCHD APRAY	OUT-OF-PHASE WHIRL AND SPIN VELOCITY SENSITIVE DAMPING FORCE COEFFICIENTS AT ROTOR STATIONS, (LB-SEC**2)/(IN-RAD)
0.0	0.0 0.0 0.0 0.0
QKF ARRAY	IN-PHASE STIFFNESS MOMENT COEFFICIENTS AT ROTOR STATIONS, IN-LB.
0.0	0.0 0.0 0.0 0.0
QCF ARRAY	IN-PHASE DAMPING MOMENT COEFFICIENTS AT ROTOR STATIONS, IN-LB-SEC.
0.0	0.0 0.0 0.0 0.0
QKPF ARRAY	OUT-OF-PHASE STIFFNESS MOMENT COEFFICIENTS AT ROTOR STATIONS, IN-LB.
0.0	0.0 0.0 0.0 0.0
QCPF ARRAY	OUT-OF-PHASE DAMPING MOMENT COEFFICIENTS AT ROTOR STATIONS, IN-LB-SEC.
0.0	0.0 0.0 0.0 0.0
Q<HDF ARRAY	OUT-OF-PHASE WHIRL AND SPIN VELOCITY SENSITIVE STIFFNESS MOMENT COEFFICIENTS AT ROTOR STATIONS, (LB-IN-SEC)/RAD.
0.0	0.0 0.0 0.0 0.0
QCHDF ARRAY	OUT-OF-PHASE WHIRL AND SPIN VELOCITY SENSITIVE DAMPING MOMENT COEFFICIENTS AT ROTOR STATIONS, (LB-IN-SEC**2)/RAD.
0.0	0.0 0.0 0.0 0.0

	NB	=	2	NUMBER OF NON-LINEAR STIFFNESS BEARINGS. (ALLOWABLE RANGE: 2=<NB=<12)
	IB	ARRAY		ROTOR STATION NUMBERS FOR NON-LINEAR STIFFNESS BEARINGS
			1	5
	K	ARRAY		TOTAL NUMBER OF STIFFNESS SECTIONS FOR EACH OF THE NON-LINEAR STIFFNESS BEARINGS. (ALLOWABLE RANGE: 1=<K=<4)
			1	1
	BKMX	ARRAY		NON-ISOTROPIC MOUNT STIFFNESS COEFFICIENTS IN X-DIRECTION FOR NON-LINEAR STIFFNESS BEARINGS, LB/IN.
			8.00000 04	8.00000 04
	BKMY	ARRAY		NON-ISOTROPIC MOUNT STIFFNESS COEFFICIENTS IN Y-DIRECTION FOR NON-LINEAR STIFFNESS BEARINGS, LB/IN.
			8.00000 04	8.00000 04
T/C	RCMX	ARRAY		NON-ISOTROPIC MOUNT DAMPING COEFFICIENTS IN X-DIRECTION FOR NON-LINEAR STIFFNESS BEARINGS, (LB-SEC)/IN.
			1.00000-16	1.00000-16
	RCMY	ARRAY		NON-ISOTROPIC MOUNT DAMPING COEFFICIENTS IN Y-DIRECTION FOR NON-LINEAR STIFFNESS BEARINGS, (LB-SEC)/IN.
			1.00000-16	1.00000-16
	BCB	ARRAY		BEARING DAMPING COEFFICIENTS FOR NON-LINEAR STIFFNESS BEARINGS, (LB-SEC)/IN.
			1.00000-16	1.00000-16

NON-LINEAR BEARING SPECIFICATIONS

0NB ARRAY ROTOR SPIN-SPEED SENSITIVE BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
 SECTIONS OF NON-LINEAR STIFFNESS BEARING NUMBER 1
 0.0

0NB ARRAY ROTOR SPIN-SPEED SENSITIVE BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
 SECTIONS OF NON-LINEAR STIFFNESS BEARING NUMBER 2
 0.0

0BB ARRAY ROTOR SPIN-SPEED SENSITIVE BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
 SECTIONS OF BEARINGS FOR NON-LINEAR STIFFNESS BEARING NUMBER 1
 0.0

0BB ARRAY ROTOR SPIN-SPEED SENSITIVE BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
 SECTIONS OF BEARINGS FOR NON-LINEAR STIFFNESS BEARING NUMBER 2
 0.0

135 0B0B ARRAY UPPER BEARING-DISPLACEMENT LIMITS FOR K STIFFNESS SECTIONS OF BEARING
 NUMBER 1, IN.
 5.0000D-03

0B0B ARRAY UPPER BEARING-DISPLACEMENT LIMITS FOR K STIFFNESS SECTIONS OF BEARING
 NUMBER 2, IN.
 5.0000D-03

0KB ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
 SECTIONS OF BEARING
 NUMBER 1
 8.0000D 04

0KB ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
 SECTIONS OF BEARING
 NUMBER 2
 8.0000D 04

0HB ARRAY NON-LINEAR BEARING STIFFNESS EXPONENTS FOR K STIFFNESS SECTIONS OF BEARING
 NUMBER 1
 1.0000D 10

RHB ARRAY NON-LINEAR BEARING STIFFNESS EXPONENTS FOR K STIFFNESS SECTIONS OF BEARING
 NUMBER 2
 1.000000 00

RDB ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS SECTIONS OF BEARING
 NUMBER 1
 0.0

RDB ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS SECTIONS OF BEARING
 NUMBER 2
 0.0

REB ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS SECTIONS OF BEARING
 NUMBER 1
 0.0

REB ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS SECTIONS OF BEARING
 NUMBER 2
 0.0

INITIAL ROTOR MOTION SPECIFICATIONS

ICOND = 1 ICOND=1 MEANS STARTING A NEW ROTOR-BEARING CONFIGURATION
 ICOND=0 MEANS READ-IN ALTERNATE INITIAL CONDITIONS FOR THE SAME
 ROTOR-BEARING CONFIGURATION

CONTIN = 0 CONTIN=0 MEANS STARTING A NEW ROTORDYNAMICS ANALYSIS WITH INITIAL CONDITIONS
 PROVIDED BY THE STARTUP SUBROUTINE
 CONTIN=1 MEANS CONTINUATION OF A PREVIOUS ANALYSIS BY USING THE PREVIOUS
 RESULTS ON PUNCHED CARDS AS THE INITIAL CONDITIONS

FD = 0.0 ROTOR SPIN ANGULAR DISPLACEMENT COORDINATE, DEGREES.
 FDOT = 1.0000D 02 ROTOR SPIN FREQUENCY, RPM.
 WHIVEL = 1.0000D 02 ROTOR WHIRL FREQUENCY, RPM.
 FDDFIX = 0.0 A BEARING STIFFNESS SPEED SENSITIVE PARAMETER, RPM

.....THE END OF INPUT DATA.....

TOTAL ROTOR WEIGHT, LB = 9.4248D 01
 TOTAL ROTOR MASS, (LB*SEC**2)/IN = 2.4411D-01
 TOTAL ROTOR POLAR MASS MOMENT OF INERTIA, LB*IN*SEC**2 = 3.0514D 00

FORCE INFLUENCE COEFFICIENTS

ROW	1				
0.0		0.0	0.0	0.0	0.0
ROW	2				
0.0		4.5001100-05	5.5000730-05	2.5000360-05	0.0
ROW	3				
0.0		5.5000730-05	8.0001460-05	5.5000730-05	0.0
ROW	4				
0.0		2.5000360-05	5.5000730-05	4.5001100-05	0.0
ROW	5				
0.0		0.0	0.0	0.0	0.0

THE TABULATED FORCE INFLUENCE COEFFICIENTS ABOVE ARE THOSE WITH RESPECT TO THE STRAIGHT LINE CONNECTING THE FIRST AND LAST JOURNAL CENTERS OF THE NON-LINEAR STIFFNESS BEARINGS. THE ROW NUMBER REPRESENTS THE ROTOR STATION NUMBER WHERE A UNIT TRANSVERSE LOAD OF 1 LB IS APPLIED, AND THE COLUMN NUMBER NOT SHOWN ABOVE, DENOTES THE ROTOR STATION NUMBER WHERE THE RESULTING DEFLECTIONS IN INCHES ARE DESCRIBED. SINCE THE FORCE INFLUENCE COEFFICIENTS ARE SYMMETRIC ABOUT THE DIAGONALS, ROW AND COLUMN NUMBERS MAY BE INTERCHANGED.

MOMENT INFLUENCE COEFFICIENTS

ROW 1 0.0	5.249999D-05	5.999999D-05	3.749999D-05	0.0
ROW 2 0.0	2.999999D-05	4.499999D-05	2.999999D-05	0.0
ROW 3 0.0	-7.499999D-06	3.388137D-05	7.499999D-06	0.0
ROW 4 0.0	-2.999999D-05	-4.499999D-05	-2.999999D-05	0.0
ROW 5 0.0	-3.749999D-05	-5.999999D-05	-5.249999D-05	0.0

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THE SIGNIFICANCE OF THE TABULATED MOMENT INFLUENCE COEFFICIENTS CAN BE SIMILARLY INTERPRETED AS THAT OF THE FORCE INFLUENCE COEFFICIENTS, EXCEPT THAT A UNIT MOMENT (1 IN-LB) APPLICATION IS USED INSTEAD OF A FORCE APPLICATION. THE COLUMN NUMBER REPRESENTS THE ROTOR STATION NUMBER WHERE THE RESULTING DEFLECTIONS IN INCHES ARE DESCRIBED. SINCE THE MOMENT INFLUENCE COEFFICIENTS ARE NO LONGER SYMMETRIC ABOUT A DIAGONAL, INTERCHANGE OF THE COLUMN AND ROW IS NOT PERMISSIBLE.

ELEMENTS IN ROTATING COORDINATES:

3.3499D-07	1.2365D-06	1.4037D-06	1.2365D-06	3.3499D-07	7.3540D-01
7.8540D-01	7.8540D-01	7.3540D-01	7.8540D-01	1.1302D-22	0.0
1.2592D-22	0.0	1.5071D-22	1.0472D 01	1.0472D 01	1.0472D 01
1.0472D 01	1.0472D 01	C.C	1.0472D 01		

THE ROTATING ELEMENTS SHOWN ABOVE ARE THOSE UNDER THE INITIAL TIME CONDITIONS, i.e., AT A STARTING TIME.

THE FIRST NS ELEMENTS IN ROTATING COORDINATES REPRESENT THE ROTOR RADIAL DISPLACEMENTS AT STATIONS 1 THROUGH NS WITH RESPECT TO RESPECTIVE STATIC ROTOR DEFLECTION CENTERS IN INCHES. THE STATIC ROTOR DEFLECTION CENTERS ARE THE ROTOR POSITIONS UNDER ZERO SPIN SPEED CONDITIONS BUT INCLUDING THE EFFECTS OF ϵ_x AND ϵ_y .

THE SECOND NS ELEMENTS DENOTE THE ABSOLUTE ANGULAR DISPLACEMENTS OF THE RADIAL DISPLACEMENT VECTORS IN RADIANS.

THE THIRD NS ELEMENTS REPRESENT THE RADIAL VELOCITIES OF THE RADIAL DISPLACEMENT VECTORS IN INCHES PER SECOND.

THE FOURTH NS ELEMENTS DENOTE THE ANGULAR VELOCITIES OF THE DISPLACEMENT VECTORS IN RADIANS PER SECOND.

THE SECOND LAST ELEMENT REPRESENTS THE ROTOR SPIN ANGULAR DISPLACEMENT IN RADIANS, AND THE LAST ELEMENT DENOTES THE ROTOR SPIN VELOCITY IN RADIANS PER SECOND.

THE APPROPRIATE DERIVATIVES ARE

1.1303440-22	1.0	1.2592200-22	0.0	1.5071260-22	1.0471980 C1	1.0471980_01
1.0471980_01	1.0471980_01	1.0471980_01	-2.4586590-13	4.6145970-14	1.5382910-13	4.6153590-14
-2.4586100-13	1.4654070-11	6.0011200-11	-5.3278940-11	5.9691000-11	3.1145690-11	1.0471980_01
-1.4060930-21						

THE BD'S LISTED ABOVE ARE THE CORRESPONDING TIME (SECOND) DERIVATIVES OF THE PREVIOUSLY LISTED ELEMENTS IN ROTATING COORDINATES. THESE BD'S ARE EVALUATED AT THE LAST SPECIFIED REAL TIME.

AT THE TIME: T = 1.025625D-03 SEC.

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)
2.3205D-07 8.5652D-07 1.1109D-06 8.5652D-07 2.3205D-07

Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)
2.4160D-07 8.9178D-07 1.1567D-06 8.9178D-07 2.4160D-07

ROTOR DEFLECTION VECTOR ARRAY (IN.)
3.3499D-07 1.2365D-06 1.6037D-06 1.2365D-06 3.3499D-07

PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES
4.6155D 01 4.6155D 01 4.6155D 01 4.6155D 01 4.6155D 01

X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN.)
-2.5300D-06 -9.3387D-06 -1.2112D-05 -9.3387D-06 -2.5300D-06

112 Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN.)
2.4300D-06 8.9695D-06 1.1634D-05 8.9695D-06 2.4300D-06

WHIRL FREQUENCY ARRAY, RPM
1.0000D 02 1.0000D 02 1.0000D 02 1.0000D 02 1.0000D 02

WHIRL TO SPIN FREQUENCY RATIO ARRAY
1.0000D 00 1.0000D 00 1.0000D 00 1.0000D 00 1.0000D 00

TOTAL NO. OF REVOLUTIONS = 3.2094D-03
SPIN SPEED, RPM = 1.0000D 02

XB ARRAY BEARING DISPLACEMENT COORDINATE, IN.
1.16020-07 1.16020-07

YB ARRAY BEARING DISPLACEMENT COORDINATE, IN.
1.20800-07 1.20800-07

XBDOT ARRAY BEARING VELOCITY COORDINATE, IN/SEC.
-1.26500-06 -1.26500-06

YBDOT ARRAY BEARING VELOCITY COORDINATE, IN/SEC.
1.21500-06 1.21500-06

BRGPD ARRAY JOURNAL DISPLACEMENT FROM BEARING-CENTER ARRAY, IN.
1.67490-07 1.67490-07

PHABRO ARRAY JOURNAL DISPLACEMENT VECTOR PHASE ANGLE COUNTER-CLOCKWISE FROM X-AXIS, DEGREES
4.61550 01 4.61550 01

BRGFOP ARRAY BEARING REACTION ARRAY, LB.
1.34000-02 1.34000-02

PHAFOR ARRAY BEARING REACTION VECTOR PHASE ANGLE COUNTER-CLOCKWISE FROM X-AXIS, DEGREES
4.61550 01 4.61550 01

STIFF ARRAY BEARING FORCE TO JOURNAL DEFLECTION RATIO, OR EQUIVALENT LINEAR BEARING
STIFFNESS, LB/IN/BEARING
8.00000 04 8.00000 04

THE APPROPRIATE DERIVATIVES BD,S

-3.4016710-16	1.8139530-16	3.8975500-16	1.8141510-16	-3.4015720-16	1.0471980 01	1.0471980 01
1.0471980 01	1.0471980 01	1.0471980 01	-7.1133110-14	1.6734710-13	2.7484970-13	1.6734720-13
-7.1123030-14	-3.7594490-09	-1.6798910-09	-1.6292380-09	-1.6785420-09	-3.7661610-09	1.0471980 01
-1.2113550-20						

APPENDIX B

COMPUTER PROGRAM LISTINGS

A complete set of the computer program Fortran Listing is attached. This includes also three library subroutines "ISIMDD," "TIMEV" and COUNTV" which may not be available in all the computation facilities. Subroutine CONTV is a part of TIMEV subroutine.

The main program and subroutine names and corresponding memory length required are:

<u>NAME</u>	<u>*MEMORY REQUIRED IN HEXADECIMAL BYTES</u>
MAIN	2EC2
READIN	D34
WRIØUT	38AC
STARJUP	1581A
FUND	2EEA
BRGXY	9EE
INFLCØ	B12
SZMASS	5F8
PLØT1	29A
RKADAM	3CC3
RUNKUT	6DE
ADAMLT	D2C
TIME	20C
ISIMDD	7D6
TIMEV	254

*The memory capacity required for "MAIN" "WRIØUT" and "PLØT1" may slightly deviate from the exact values resulting from changing to NASA CRT Subroutines and minor modifications after memory capacity counts.

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C      A SIMPLIFIED FLEXIBLE ROTOR DYNAMICS COMPUTER PROGRAM WRITTEN FOR 00768000
C      NASA LEWIS RESEARCH CENTER, CLEVELAND, OHIO, BY FRED. A. SHEN,    00769000
C      POWER SYSTEMS DIVISIONS, NORTH AMERICAN ROCKWELL CORP., LOS ANGELES 007000
C      CALIFORNIA, MAY 1970.                                             00771000
      IMPLICIT REAL*8 (A-H,O-Z)                                         00772000
      INTEGER CRF,ACCEL,COUNTIN                                          00773000
      REAL TITLE,INPRPM,Z,ROU,TT,RPM,WHRATI,FORC,BRGR,ROMAX,RUSTA      00774000
1,RSTAR4,RSFAR5,RSTAR6                                                00775000
      DIMENSION DD(25),D(25),QL(25),DN(25),                               AM(25),AID(25)00776000
1),AIRU(25),ECC(25),ALFA(25),BETA(25),GAMMA(25)                       00777000
2,TITLE(36),YSAVE(102),SP(3)                                           00778000
      DIMENSION QK(25),QC(25),QKP(25),QCP(25),QKHD(25),QCHD(25),QKF(25)00779000
1),QCF(25),QKPF(25),QCPF(25),QKHDF(25),QCHDF(25)                       00782000
2,IB(12),K(25)                                                         00783000
3,BKMX(25),BKMY(25),BCMX(25),BCMY(25),BCB(25),                          RU(25)    00784000
4,BNB(25,6),BBB(25,6),                                                BRUB(25,7), 00785000
5C(25,25),B(25,25)                                                    00786000
      DIMENSION XX(25),YY(25),XDU(25),YDU(25)                           00789000
1,XB(25),YB(25),XBDOT(25),YBDOT(25)                                    00790000
1,QM(25),DELTX(25),DELTY(25),WHRVLD(25),WHIRPM(25),Y(102),KK(25),     00793000
2BRGRU(25),PHABRU(25),BRGFUR(25),PHAFUR(25),WHRATI(50),STIFF(25)     00794000
4,RUSQ(25),WHRATU(25),ROMAX(50),ISTATN(50),RPM(50),RUSFA(50),        INPR00797000
5PM(50),ROU(25),PHAROU(25),Z(25),BRGR(50,12),FORC(50,12),TT(50),    00798000
6SZ(25),ZQ(25),SZUL(25),ZQL(25),ZQL(25),QZ(25),ZSUL(25),QZUL(25)    00799000
      COMMON/MAREAD/TITLE,INPRPM,                                       I,DT,TMAX,  TMZ,IMZ1,FMZ2,FMZ300803000
1,      TOLI, NOURPM,          IASIGN,CRF,ACCEL                          00804000
      COMMON/MAFUF1/DD,D,QL, NS,NB,IB                                   00806000
      COMMON/MAFUF2/ SZ,QZ,ZQ,SZUL,ZSUL,QZUL,ZQL,QMLOV,QLL             00807000
      COMMON/MAFUF3/ IB1,IBNB                                          00808000
      COMMON/MAFUF4/ Z                                                 00809000
1      /MAFUO/ITIM,INT,KK                                             00810000
      COMMON/MAFU14/ FDUFIX                                             00810500
      COMMON/MAFU1/          TSTOP,K                                   00811000
      COMMON/MAFU2/ DN,AM,AID,AIRU,ECC,ALFA,BETA,GAMMA,GX,GY          00812000
1      /MAFU3/WHIVEL                                                  00813000
      COMMON/MAFU5/ QK,QC,QKP,QCP,QKHD,QCHD,QKF,QCF,QKPF,QCPF,QKHDF,QC00815000
1HDF, BKMX,BKMY,BCMX,BCMY,BCB                                         00816000
      COMMON/MAFU6/ BNB,BBB,BRUB                                       00817000
      COMMON /MAFU7/RUSQ, WHRVLU,DELTX,DELTY                           00818000
      COMMON/MAFU9/ C,B                                                00820000
1      /MAFU10/RU,PHAROU                                              00821000
      COMMON/MAFU11/FD,FUUF                                           00822000

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SHI

	1	/MAFU13/Y	00823000
	X	/SFP1/RPM,NPOINT,ICOND,CUNFIN	00824000
	1	/SFP2/NSM1,NSM2,NSM3,NSP1,NS2,NS3,NS4,NS4P1,N,NS2P1	00824100
	1	/GARBG7/XX,YY,XB,YB,XDOT,YDOT,XBDOT,YBDOT	00824200
	2	/GARBG9/YSAVE,WHIRPM,BRGRO,PHABRO,BRGFOR,PHAFOR,WHRATO,	00824300
	3	PHARU(25),RUMAX,ISTATN,RUD,BRGR,FORC,GT,WHRATI	00824400
		COMMON/MAFU12/ QM	00825000
		DATA PI/3.14159265358979324/	00825500
404		FORMAT (7X,1P6E13.4)	00826000
		A=180/PI-	00826100
		H=30/PI	00826200
		V=1/(2*PI)	00826300
		GU TU 2	00826400
	1	KA=0	00826410
		DO 5 I=1,N,3	00826420
		SP(1)=Y(I)	00826430
		SP(2)=Y(I+1)	00826440
		SP(3)=Y(I+2)	00826450
		KA=KA+1	00826460
	5	PUNCH 106,SP,KA	00826470
106		FORMAT(1P3D22.15,6X,I8)	00826480
	2	CALL COUNTV	00826700
		TMIN=0	00826710
9TT		ICC=0	00826800
		CALL READIN	00828000
		CALL WRIOU	00829000
		TR=T	00829500
		IC=0	00830000
		ITIM=1	00831000
		IND=0	00832000
		INT=0	00833000
		MM=1	00834000
		MAXSHF=0	00836000
		WHIVEL=WHIVEL/H	00837000
		IF(ICOND.EQ.0),GO TU 3	00838000
		DO 33 I=1,NB	00845000
		IBI=IB(I)	00846000
33		K<{ BI}=1	00847000
		IB1=IB(1)	00857000
		IBNB=IB(NB)	00858000
		CALL SZMASS	00859000
		CALL INFLCO(C,B)	00860000

	DU 22 J=1,NS	00861000
	DELT=0.	00862000
	DU 23 I=1,NS	00863000
23	DELT=DELT+C(I,J)*QM(I)	00865000
	DELTX(J)=DELT*GX	00866000
22	DELTY(J)=DELT*GY	00867000
3	ICOND=0	00867050
	IF(CONTIN.EQ.1)GOTO80	00867100
	GO TO 8	00867110
7	ITIM=1	00867120
	DT=0.1*DT	00867130
8	CALL STARUP	00867200
	DO 95 I=1,NS	00868000
	INS=I+NS	00869000
	I2NS=I+NS2	00870000
	I3NS=I+NS3	00871000
	PHAROU(I)=PHAROU(I)/A	00871100
	XNUTR=RO(I)*DCOS(PHAROU(I))	00871200
	YNUTR=RO(I)*DSIN(PHAROU(I))	00871300
	XDOT(I)=-WHIVEL*YNUTR	00871400
	YDOT(I)=WHIVEL*XNUTR	00871500
	Y(I)=DSQRT(XNUTR**2+YNUTR**2)	00874000
	Y(INS)=DATAN(YNUTR/XNUTR)	00875000
	Y(I2NS)=(XNUTR*XDOT(I)+YNUTR*YDOT(I))/Y(I)	00876000
95	Y(I3NS)=(XNUTR*YDOT(I)-YNUTR*XDOT(I))/Y(I)**2	00877000
	Y(NS4P1)=FD/A	00879000
	Y(N)=FDOT/H	00880000
80	WRITE(6,1555)	00880100
1555	FORMAT(1H04X'ELEMENTS IN ROTATING COORDINATES:')	00880200
	WRITE(6,404)(Y(I),I=1,N)	00880300
100	IF(CRT.EQ.0) GO TO 107	00881000
	IF(T.GE.TMAX.OR.TMIN.GE.TSTOP) GO TO 1040	00881500
	PPM=0	00882000
	P1=FDOT	00883000
107	TSAVE=1	00884000
	DU 41 I=1,N	00885000
41	YSAVE(I)=Y(I)	00886000
24	IF(T.GE.TMAX) GO TO 900	00887000
	ICC=ICC+1	00887100
C	IF(ICC.LI.10) GO TO 97	00887200
	CALL TIMEV(TSEC)	00887300
	TMIN=TSEC/60	00887400

	IF(TMIN.GE.TSTOP) GO TO 1	00887500
	ICC=0	00887600
97	CALL RKADAM(N,TR,Y,DT,IND,ITIM,TOLI,IERR)	00888000
	DO 4 I=1,N	00888100
	IF(Y(I).GE.1.D14) GO TO 7	00888200
4	CONTINUE	00888300
	CALL TIME(TR,DT,T,ITIM)	00888600
	IF(IERR.EQ.0)GOTO108	00889000
	WRITE(6,311)T	00890000
311	FORMAT('-UNSUCCESSFUL SOLUTION AT T ='1PD12.5)	00891000
	STOP	00892000
108	IF(CRT.EQ.0) GO TO 110	00893000
	PM1=INPRPM(MM)-P1*H	00895000
	PM2=INPRPM(MM)-H*Y(N)	00896000
	P12=PM1*PM2	00897000
	IF(P12.GT.0) GO TO 110	00898000
	MM=MM+1	00899000
	IF(DABS(PM2).GT.DABS(PM1))GOTO1000	00900000
110	DO 32 I=1,NS	00904000
	INS=I+NS	00905000
	I2NS=I+NS2	00906000
	I3NS=I+NS3	00907000
	COSAB=DCOS(Y(INS))	00908000
	SINAB=DSIN(Y(INS))	00909000
	XNUTR=Y(I)*COSAB	00910000
	YNUTR=Y(I)*SINAB	00911000
	XX(I)=XNUTR-DELTX(I)	00912000
	YY(I)=YNUTR-DELY(I)	00913000
	PHARO(I)=Y(INS)	00914000
	XDDOT(I)=Y(I2NS)*COSAB-Y(I)*Y(I3NS)*SINAB	00915000
	YDDOT(I)=Y(I2NS)*SINAB+Y(I)*Y(I3NS)*COSAB	00916000
32	WHRVLO(I)=Y(I3NS)	00917000
	FDDT=Y(N)	00922000
	CALL BRGXY	00923000
	A1=0	00924000
	B1=0	00925000
	DO 317 I=1,NB	00926000
	IBI=IB(I)	00927000
	BRGR0(IBI)=DSQRT(XB(IBI)**2+YB(IBI)**2)	00928000
	KP1=K(IBI)+1	00929000
	IF(BRGR0(IBI).GT.BROB(IBI,KP1)) GO TO 27	00930000
	KKB=KK(IBI)	00931000

25	IF(BRGRU(IBM).LT.BROB(IBM,KKB))GOTO29	00932000
26	KKB1=KKB+1	00933000
	IF(BRGRU(IBM).LE.BROB(IBM,KKB1))GO TO 28	00934000
	KKB=KKB+1	00935000
	A1=A1+1	00936000
	GO TO 26	00936700
27	WRITE(6,200)	00938000
200	FORMAT(' BEARING DEFLECTION EXCEEDS BEARING CLEARANCE')	00939000
	GO TO 1	00940000
29	KKB=KKB-1	00941000
	B1=B1+1.	00942000
	GO TO 25	00943000
28	KK(IBM)=KKB	00944000
317	CONTINUE	00945000
	A1B1=A1+B1	00947000
	IF(A1B1.EQ.0) GO TO 318	00948000
	ITIM=1	01034000
	T=TSAVE	01035000
	TR=T	01035100
	DO 102 I=1,N	01036000
102	Y(I)=YSAVE(I)	01037000
	MAXSHF=MAXSHF+1	01038000
	IF(MAXSHF.GE.10) GO TO 320	01039000
	GOTO24	01040000
320	WRITE(6,330)	01041000
330	FORMAT(1H0,4X,'THE MAXIMUM NUMBER OF SHIFTING OF STIFFNESS SECTION	01042000
	1S IS TENTATIVELY LIMITED TO 10'/4X,' TO AVOID INFINITE CYCLIC COMPU	01043000
	2TATIONS DUE TO INADEQUATE INPUT DATA.')	01044000
	GO TO 1	01045000
318	IC=IC+1	01046000
	ITIM=0	01047000
	DU 43 I=1,NB	01048000
	IBM=IB(I)	01049000
	BRGRU(IBM)=DSQRT(XB(IBM)**2+YB(IBM)**2)	01050000
	PHABRU(IBM)=DATN2D(YB(IBM),XB(IBM))	01051000
	IF(PHABRU(IBM).LT.0) PHABRU(IBM)=360.+PHABRU(IBM)	01052000
	BRGFOX=BKMX(IBM)*(XX(IBM)-XB(IBM))	01053000
	BRGFOY=BKMY(IBM)*(YY(IBM)-YB(IBM))	01054000
	BRGFUR(IBM)=DSQRT(BRGFOX**2+BRGFOY**2)	01055000
	STIFF(IBM)=BRGFUR(IBM)/BRGRU(IBM)	01055100
	BRGFOX=BRGFOX+XDUT(IBM)*BCB(IBM)	01055200
	BRGFOY=BRGFOY+YDUT(IBM)*BCB(IBM)	01055300

	BRGFUR (IBI)=DSQRT (BRGFOX**2+BRGFQY**2)	01055400
	PHAFUR (IBI)=DATN2D (BRGFQY, BRGFOX)	01056000
43	IF (PHAFUR (IBI).LT.0) PHAFOR (IBI)=360.+PHAFUR (IBI)	01057000
	FDDT=Y(N)	01059000
	REVULN=V*Y (NS4P1)	01060000
	DO 45 I=1,NS	01061000
	WHIRPM(I)= H*WHRVLO(I)	01062000
45	WHRATO(I)=WHRVLO(I)/FDDT	01063000
	WHRATI (IC)=WHRATO (IASIGN)	01064000
	RPMM= H*FDDT	01065000
	RPM(IC)=RPMM	01066000
	TT(IC)=T	01067000
	WRITE(6,1001)T	01068000
1001	FORMAT (1H1///34X, 16HAT THE TIME: T=1PD13.6,2X,'SEC.')	01069000
	WRITE (6,701)	01070000
701	FORMAT(1H0,4X,52HX DISPLACEMENT ARRAY IN STATIONARY COORDINATES,(1	01071000
	IN.))	01072000
	WRITE (6,404) (XX(I),I=1,NS)	01073000
	WRITE (6,702)	01074000
702	FORMAT(1H0,4X,52HY DISPLACEMENT ARRAY IN STATIONARY COORDINATES,(1	01075000
	IN.))	01076000
150	WRITE (6,404) (YY(I),I=1,NS)	01077000
	WRITE(6,707)	01078000
707	FORMAT(1H0,4X,35H ROTOR DEFLECTION VECTOR ARRAY (IN.))	01079000
	DO 34 I=1,NS	01080000
	RO(I)=Y(I)	01081000
	PHARO(I)=DATN2D(YY(I),XX(I))	01082000
34	IF (PHARO(I).LT.0) PHARO(I)=360.+PHARO(I)	01083000
	WRITE(6,404) (RO(I),I=1,NS)	01084000
	WRITE(6,708)	01085000
708	FORMAT(1H0,4X, 'PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DO	01086000
	1EGRES')	01087000
	WRITE(6,404) (PHARO(I),I=1,NS)	01088000
	WRITE (6,703)	01089000
703	FORMAT(1H0,4X, 'X VELOCITY ARRAY IN STATIONARY COORDINATES,(IN/SE01090000	01090000
	1C)')	01091000
	WRITE (6,404) (XDOT(I),I=1,NS)	01091000
	WRITE (6,704)	01092000
704	FORMAT(1H0,4X, 'Y VELOCITY ARRAY IN STATIONARY COORDINATES,(IN/SE01093000	01093000
	1C)')	01093100
	WRITE (6,404) (YDOT(I),I=1,NS)	01094000
	WRITE (6,705)	01095000

705	FORMAT (1H0,4X,25HWHIRL FREQUENCY ARRAY,RPM)	01096000
	WRITE (6,404) (WHIRPM(I),I=1,NS)	01097000
	WRITE (6,706)	01098000
706	FORMAT (1H0,4X,35HWHIRL TO SPIN FREQUENCY RATIO ARRAY)	01099000
	WRITE (6,404) (WHRATO(I),I=1,NS)	01100000
	WRITE(6,1002)REVOLN,RPMM	01101000
1002	FORMAT(//5X,26HTOTAL NO. OF REVOLUTIONS =1PD13.4/ 5X,26HSPIN SPEED	01102000
	1D, RPM =1PD13.4)	01103000
	WRITE (6,457)	01104000
457	FORMAT(//1H1//5X,8HXB ARRAY,10X, 36HBEARING DISPLACEMENT COORDINATE	01105000
	1E, IN.)	01106000
	WRITE (6,404) (XB(IB(I)),I=1,NB)	01107000
	WRITE (6,458)	01108000
458	FORMAT (1H0,4X,8HYB ARRAY,10X,36HBEARING DISPLACEMENT COORDINATE,	01109000
	1IN.)	01110000
	WRITE (6,404) (YB(IB(I)),I=1,NB)	01111000
	WRITE (6,459)	01112000
459	FORMAT (1H0,4X,11HXBDOT ARRAY,7X, 'BEARING VELOCITY COORDINATE,	01113000
	1IN/SEC.')	01114000
	WRITE (6,404) (XBDOF(IB(I)),I=1,NB)	01115000
	WRITE(6,460)	01116000
460	FORMAT (1H0,4X,11HYBDOT ARRAY,7X, 'BEARING VELOCITY COORDINATE,	01117000
	1IN/SEC.')	01118000
	WRITE (6,404) (YBDOT(IB(I)),I=1,NB)	01119000
	WRITE (6,1003)	01120000
1003	FORMAT (//1H0,4X, 11HBRGRD ARRAY,7X, 'JOURNAL DISPLACEMENT FROM	01121000
	1BEARING-CENTER ARRAY, 'IN.')	01122000
	WRITE (6,404) (BRGRD(IB(I)),I=1,NB)	01123000
	WRITE (6,1010)	01124000
1010	FORMAT (1H0,4X,12HPHABRD ARRAY,7X, 'JOURNAL DISPLACEMENT VECTOR	01125000
	1PHASE ANGLE COUNTER-CLOCKWISE FROM X-AXIS, DEGREES')	01126000
	WRITE (6,404) (PHABRO(IB(I)),I=1,NB)	01127000
	WRITE (6,1008)	01128000
1008	FORMAT(1H0,4X,12HBRGFOR ARRAY,6X,'BEARING REACTION ARRAY, LB.')	01129000
	WRITE (6,404) (BRGFOR(IB(I)),I=1,NB)	01130000
	WRITE (6,1009)	01131000
1009	FORMAT (1H0,4X, 12HPHAFOR ARRAY, 6X, 'BEARING REACTION VECTOR PHO	01132000
	1ASE ANGLE COUNTER-CLOCKWISE FROM X-AXIS, DEGREES')	01133000
	WRITE (6,404) (PHAFOR(IB(I)),I=1,NB)	01134000
	WRITE(6,1020)	01134100
1020	FORMAT(1H0,4X,11HSTIFF ARRAY,7X'BEARING FORCE TO JOURNAL DEFLECTIO	01134200
	1N RATIO, OR EQUIVALENT LINEAR BEARING'/24X,'STIFFNESS, LB/IN/BEAR	01134300

151

	2NG')		01134310
	WRITE(6,404) (STIFF(IB(I)),I=1,NB)		01134400
	IF(CRT.NE.0) GO TO 105		01135000
	IC=1		01135100
	GO TO 107		01135200
105	DO 500 I=1,NB		01136000
	J=IB(I)		01137000
	FORC(IC,I)=BRGFOR(J)		01138000
500	BRGR(IC,I)=BRGRU(J)		01139000
	IF(ACCEL.EQ.0)GOTO1000		01140000
	IF(P12.GT.0.0) GO TO 1027		01141000
	GO TO 1000		01142000
900	IF(CRT.EQ.0.OR.ACCEL.NE.0)GOTO1		01143000
1000	ACCEL =1		01145000
	DO 1104 I=1,NS		01146000
1104	ROO(I) =RO(I)		01147000
C	PLOT 1		01147005
	REAL CHAR11(21),CHAR21(7),CHAR31(8),CHARSS(4),SYMBOL/'*'/		01147010
	DATA CHAR11/'ROTOR 3-DIMENSIONAL MODE SHAPE WITH PHASE ANGLES (DEG	01147020	
	1REES) LABELED AS SHOWN, AT RPM=/',CHAR21/'ROTOR AXIAL LENGTH, INC	01147030	
155	2HES/',CHAR31/'ROTOR DEFLECTIION VECTOR, INCHES'/		01147040
	CALL LRLEGN(TITLE,72,0,1.15,9.99,0.)		01147050
	CALL LRLEGN(TITLE(19),72,0,1.15,9.83,0.)		01147060
	CALL LRLEGN(CHAR11,84,0,1.463,9.67,0.)		01147070
	CALL LRCNVT(RPMM,3,CHARS ,4,13,5)		01147080
	CALL LRLEGN(CHARSS,13,0,8.6,9.67,0.)		01147090
	CALL LRLEGN(CHAR21,27,0,4.31,0.,0.)		01147100
	CALL LRCURV(Z,ROO,NS,2,SYMBOL,0.)		01147110
	DO 1005 'I=1,NS		01155000
	RSTAR4=PHARD(I)		01155010
	CALL LRCNVT(RSTAR4,3,CHARSS,3,4,0)		01155020
	CALL LRLABL(CHARSS,4,0,Z(I),RUO(I),0.)		01155030
1005	CONTINUE		01160000
	CALL LRLEGN(CHAR31,31,1,0.,4.6,1.)		01160010
	IF(DABS(PM2).LE.DABS(PM1)) GO TO 1027		01161000
	IF(PPM.EQ.1) GO TO 100		01162000
	PPM=1		01163000
	GO TO 110		01164000
1027	PPM=0		01165000
	J=1		01167000
	DO 1209 I=1,NS		01168000
1209	IF(RO(J).LF.RD(I))J=I		01169000

	ROMAX(IC)=RO(J)	01170000
	ISTATN(IC)=J	01171000
	RUSTA(IC)=RO(IASIGN)	01171500
	IF(IC.LT.NPOINT) GO TO 100	01171600
1040	NPOINT =IC	01172000
C	PLOT 2	01173000
	CALL LRLEGN(TITLE,72,0,1.15,9.99,0.)	01173500
	CALL LRLEGN(TITLE(19),72,0,1.15,9.83,0.)	01174000
	REAL CHAR12(7) / ROTOR SPIN SPEED VERSUS TIME',CHAR22(4) / TIME, SE01174500	
1	CONDS ' / ,CHAR32(6) / ROTOR SPIN SPEED, RPM ' /	01175000
	CALL LRCURV(TT,RPM,NPOINT,2,SYMBOL,0.)	01175500
	CALL LRCURV(TT,RPM,NPOINT,3,SYMBOL,0.)	01176000
	CALL LRLEGN(CHAR12,28,0,3.50,9.67,0.)	01176500
	CALL LRLEGN(CHAR22,13,0,4.86,0.,0.)	01177000
	CALL LRLEGN(CHAR32,21,1,0.,4.95,1.)	01177100
	IF(ACCEL.EQ.0)GOTO2000	01177200
C	PLOT 3	01177400
	CALL LRLEGN(TITLE,72,0,1.15,9.99,0.)	01177500
	CALL LRLEGN(TITLE(19),72,0,1.15,9.83,0.)	01178000
	REAL CHAR13(19) / ROTOR WHIRL-TO-SPIN FREQUENCY RATIO VERSUS ROTOR	01178500
153	SPIN SPEED AT ROTOR STATION' /	01179000
	CALL LRCURV(RPM,WHRATI,NPOINT,2,SYMBOL,0.)	01179500
	CALL LRCURV(RPM,WHRATI,NPOINT,3,SYMBOL,0.)	01180000
	CALL LRCNV(IASIGN,1,CHARSS,1,3,0)	01181000
	CALL LRLEGN(CHARSS,3,0,7.85,9.67,0.)	01181500
	CALL LRLEGN(CHAR13,76,0,1.795,9.67,0.)	01182000
	CALL LRLEGN(CHAR13,35,1,0.,3.47,0.)	01182500
	CALL LRLEGN(CHAR32,21,0,4.55,0.,1.)	01183000
C	PLOT 4	01184000
C	PLOT RPM VS FORC FUNCTIONS	01185000
	CALL PLOT1(FORC)	01186000
	CALL LRLEGN(TITLE,72,0,1.15,9.99,0.)	01186100
	CALL LRLEGN(TITLE(19),72,0,1.15,9.83,0.)	01186200
	REAL CHAR14(24) / BEARING REACTIONS VERSUS ROTOR SPIN SPEED WITH BE01186300	
	1ARING LOCATION STATION NUMBERS LABELED AS SHOWN',CHAR24(7) / BEAR01186400	
	ING REACTIONS, POUNDS ' /	01186500
	CALL LRLEGN(CHAR14,96,0,1.385,9.67,0.)	01186600
	CALL LRLEGN(CHAR32,21,0,4.55,0.,0.)	01186700
	CALL LRLEGN(CHAR24,25,1,0.,4.37,1.)	01186800
C	PLOT 5	01192000
C	PLOT RPM VS BRGR FUNCTIONS	01193000
	CALL PLOT1(BRGR)	01194000

	CALL LRLEGN(TITLE,72,0,1.15,9.99,0.)	01194100
	CALL LRLEGN(TITLE(19),72,0,1.15,9.83,0.)	01194200
	REAL CHAR15(25) //' JOURNAL DISPLACEMENT VERSUS ROTOR SPIN SPEED WITH	01194300
	1 BEARING LOCATION STATION NUMBERS LABELED AS SHOWN '/',CHAR25(8) /	01194400
	2 ' JOURNAL DISPLACEMENTS, INCHES '	01194500
	CALL LRLEGN(CHAR15,99,0,1.21,9.67,0.)	01194600
	CALL LRLEGN(CHAR32,21,0,4.55,0.,0.)	01194700
	CALL LRLEGN(CHAR25,29,1,0.,4.52,1.)	01194800
C	PLOT 6	01200000
	CALL LRANGE(0.,0.,0.,0.)	01200005
	REAL CHAR16(13) //' MAXIMUM ROTOR DEFLECTIONS VERSUS ROTOR SPIN SPEED	01201000
	1 '/',CHAR26(22) //' (THE STATION NUMBERS WHERE THE MAXIMUM DEFLECTIO	01202000
	2 NS OCCUR ARE SHOWN) '/',CHAR36(9) //' MAXIMUM ROTOR DEFLECTIONS, INCHE	01203000
	3 S '	01204000
	CALL LRCURV(RPM,RUMAX,NPOINT,2,SYMBOL,0.)	01205000
	DO 1006 I=1,NPOINT	01206000
	CALL LRCNVT(ISTATN(I),1,CHARSS,1,3,0)	01207000
1006	CALL LRLABL(CHARSS,3,0,RPM(I),ROMAX(I),0.)	01208000
	CALL LRLEGN(TITLE,72,0,1.15,9.99,0.)	01209000
	CALL LRLEGN(TITLE(19),72,0,1.15,9.873,0.)	01210000
151	CALL LRLEGN(CHAR16,49,0,3.45,9.756,0.)	01211000
	CALL LRLEGN(CHAR26,67,0,2.75,9.639,0.)	01212000
	CALL LRLEGN(CHAR32,21,0,4.55,0.,0.)	01213000
	CALL LRLEGN(CHAR36,33,1,0.,4.52,1.)	01214000
C	PLOT 7	01214010
	REAL CHAR17(17) //' (THE STATION NUMBER WHERE THE ROTOR DEFLECTIONS	01214020
	1 OCCUR IS SHOWN) /	01214030
	CALL LRCURV(RPM,ROSTA,NPOINT,2,SYMBOL,0.)	01214040
	DO 1007 I=1,NPOINT	01214050
	CALL LRCNVT(IASIGN,1,CHARSS,1,3,0)	01214060
1007	CALL LRLABL(CHARSS,3,0,RPM(I),ROSTA(I),0.)	01214070
	CALL LRLEGN(TITLE,72,0,1.15,9.99,0.)	01214080
	CALL LRLEGN(TITLE(19),72,0,1.15,9.873,0.)	01214090
	CALL LRLEGN(CHAR16(3),41,0,3.45,9.756,0.)	01214100
	CALL LRLEGN(CHAR17,63,0,2.75,9.639,0.)	01214110
	CALL LRLEGN(CHAR32,21,0,4.55,0.,0.)	01214120
	CALL LRLEGN(CHAR36(3),25,1,0.,4.25,1.)	01214130
2000	IC=0	01215000
	IF(T.GE.TMAX.OR.TMIN.GE.TSTOP) GO TO 1	01215100
	GOTO100	01216000
	END	01217000

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SUBROUTINE READIN                                01218000
IMPLICIT REAL*8 (A-H,O-Z)                       01219000
INTEGER CRT,ACCEL,CONTIN                         01220000
REAL TITLE,INPRPM, RPM                          01221000
    DIMENSION DD(25),D(25),QL(25),DN(25),EE(25),GG(25),AM(25),AID(25)01222000
1),AIRO(25),ECC(25),ALFA(25),BETA(25),GAMMA(25), EI(25),GAK(25) 01223000
2),TITLE(36),INPRPM(50),RPM(50)                 01224000
    DIMENSION CZ(25),CZ1(25),CZ2(25)           01225000
    DIMENSION XKF(25),XCF(25),XKFF(25),XCFF(25) 01226000
    DIMENSION QK(25),QC(25),QKP(25),QCP(25),QKHD(25),QCHD(25),QKF(25)01227000
1),QCF(25),QKPF(25),QCPF(25),QKHDF(25),QCHDF(25) . 01228000
2),IB(12),K(25)                                01229000
3),BKM(25),BKMY(25),BCMX(25),BCMY(25),          BCB(25), 01230000
4)BNB(25,6),BBB(25,6),BROB(25,7),BKB(25,6),BHB(25,6),BDB(25,6), 01231000
5)BEB(25,6), Y(102)                             01232000
COMMON/MAREAD/ TITLE,INPRPM, F,DT,TMAX, TMZ,TMZ1,TMZ2,TMZ3 01234000
1, TOLI, NOORPM, IASIGN,CRT,ACCEL               01235000
1 /SFP1/RPM,NPOINT,ICOND,CONTIN                 01236000
COMMON/MAFC/ EE,GG, EI,GAK                     01237000
COMMON/MAFUF1/DD,D,QL, NS,NB,IB                01238000
COMMON/MAFU14/ FDOFIX                           01238500
COMMON/MAFU1/ TSTOP,K                          01239000
COMMON/MAFU2/ DN,AM,AID,AIRO,ECC,ALFA,BETA,GAMMA,GX,GY 01240000
1 /MAFU3/WHIVEL                                 01241000
COMMON/MAFU4/ CZ,CZ1,CZ2, XKF,XCF,XKFF,XCFF    01242000
COMMON/MAFU5/ QK,QC,QKP,QCP,QKHD,QCHD,QKF,QCF,QKPF,QCPF,QKHDF,QC01243000
1HDF, BKM(25),BKMY(25),BCMX(25),BCMY(25),BCB 01244000
COMMON/MAFU6/ BNB,BBB,BROB,BKB,BHB,BDB,BEB    01245000
COMMON/MAFU8/ TOLB                              01246000
1 /MAFU13/Y                                     01247000
COMMON/MAFU11/FD,FDOT                          01248000
1 /SFP2/NSM1,NSM2,NSM3,NSP1,NS2,NS3,NS4,NS4P1,NS4P2,NS2P1 01248100
NAMESLIST/DATA1/NS, TMAX,TSTOP,DD,QL,NB,IB,K,ICOND,CONTIN, FDOT,W01249000
1HIVEL                                          01249100
NAMESLIST/DATA2/ T,FD,DT,FDOFIX,              01250000
1 IASIGN,NOORPM,NPOINT,CRT,ACCEL,INPRPM,TOLI,TOLB,GX,01251000
1GY,TMZ,TMZ1,TMZ2,TMZ3, D,DN,EE,GG, EI,GAK,AM,AID,AIRO,ECC,ALFA01252000
2),BETA,GAMMA,CZ,CZ1,CZ2,XKF,XCF,XKFF,XCFF,QK,QC,QKP,QCP,QKHD,QCHD,Q01253000
3KF,QCF,QKPF,QCPF,QKHDF,QCHDF, BKM(25),BKMY(25),BCMX(25),BCMY(25),BCB,BNB,BBB,BRO01254000
4B,BKB,BHB,BDB,BEB                             01255000
READ(5,5) TITLE                                01257000
5 FORMAT(18A4)                                  01258000

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READ(5,DATA1)	01259000
IF(NS.LT.5) GO TO 12	01261000
GO TO 11	01262000
12 WRITE (6,377)	01263000
377 FORMAT(///1H0,10X, 47HMINIMUM ALLOWABLE NUMBER OF ROTOR STATIONS	01264000
1S 5)	01265000
CALL EXIT	01266000
11 FD=0	01266500
IASIGN=1	01267000
NOORPM=1	01268000
NPUINT=25	01269000
CRT=0	01270000
ACCEL=0	01271000
INPRPM(1)=0	01272000
T=0	01273000
DT=.00001	01273500
TOLI=.01	01274000
TOLB=.001	01275000
GX=0	01276000
GY=0	01277000
TMZ=0	01278000
TMZ1=0	01279000
TMZ2=0	01280000
TMZ3=0	01281000
NSM1=NS-1	01281200
NSM2=NS-2	01281400
NSM3=NS-3	01281600
NSP1=NS+1	01281800
NS2=2*NS	01282000
NS3=3*NS	01282200
NS4=4*NS	01282400
NS4P1=NS4+1	01282600
NS4P2=NS4+2	01282800
NS2P1=NS2+1	01283000
DO 14 I=1,NSM1	01283200
D(I)=0	01284000
DN(I)=.283	01285000
EE(I)=3.07	01286000
GG(I)=1.15D7	01287000
EI(I)=0	01288000
14 GAK(I)=0	01289000
DD 16 I=1,NS	01290000

AM(I)=1.D-16	01291000
AID(I)=0	01292000
AIRO(I)=1.D-16	01293000
ECC(I)=1.D-16	01294000
ALFA(I)=0	01295000
BETA(I)=0	01296000
GAMMA(I)=0	01297000
CZ(I)=0	01298000
CZ1(I)=0	01299000
CZ2(I)=0	01300000
XKF(I)=0	01301000
XCF(I)=0	01302000
XKFF(I)=0	01303000
XCFF(I)=0	01304000
QK(I)=0	01305000
QC(I)=0	01306000
QKP(I)=0	01307000
QCP(I)=0	01308000
QKHD(I)=0	01309000
QCHD(I)=0	01310000
QKF(I)=0	01311000
QCF(I)=0	01312000
QKPF(I)=0	01313000
QCPF(I)=0	01314000
QKHDF(I)=0	01315000
QCHDF(I)=0	01316000
	01317000
DO 20 N1=1,NB	01318000
I=IB(N1)	01319000
BKMX(I)=1.D10	01321000
BKMY(I)=1.D10	01322000
BCMX(I)=1.D-16	01323000
BCMY(I)=1.D-16	01324000
20 BCB(I)= 1.D-16	01325000
FDOFIX=0	01325500
DO 21 N1=1,NB	01326000
I=IB(N1)	01327000
KI=K(I)	01328000
DD 21 J=1,KI	01329000
BNB(I,J)=0	01330000
3BB(I,J)=0	01331000
3KB(I,J)=1.D6	01332000

	BHB(I,J)=1	01333000
	BDB(I,J)=0	01334000
21	BEB(I,J)=0	01335000
	DO 22 N1=1,NB	01336000
	I=IB(N1)	01337000
22	BR0B(I,1)=.005	01340000
15	FORMAT(6E12.8)	01341000
	READ(5,DATA2)	01342000
	DT=2.*DT	01343000
	DO 24 N1=1,NB	01343100
	I=IB(N1)	01343200
	KI=K(I)	01343300
	DO 24 J=1,KI	01343400
	JP1=J+1	01343500
24	BR0B(I,JP1)=BR0B(I,J)	01343600
	DO 26 N1=1,NB	01343700
	I=IB(N1)	01343800
26	BR0B(I,1)=0	01343900
	DD(NS)=1.	01344000
	D(NS)= 0.	01345000
	QL(NS)=1.	01346000
	DN(NS)=0.	01347000
	EE(NS)=1.	01348000
	GG(NS)=1.	01349000
	EI(NS)=0.	01350000
	GAK(NS)=0.	01351000
	IF(CONTIN.EQ.0) GO TO 17	01352000
	READ(5,410) (Y(I),I=1,NS4P2)	01355000
410	FORMAT(3D22.15)	01356000
17	DO 10 I=1,NS	01356100
	IF(EI(I).NE.0) EE(I)=0	01356200
	IF(GAK(I).NE.0) GG(I)=0	01356300
10	CONTINUE	01356400
	RETURN	01367000
	END	01368000

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SUBROUTINE WRIOUT                                01570000
IMPLICIT REAL*8 (A-H,O-Z)                       01571000
INTEGER CRT,ACCEL,CONTIN                        01572000
REAL TITLE, INPRPM,RPM                          01573000
DIMENSION DD(25),D(25),QL(25),DN(25),EE(25),GG(25),AM(25),AID(25)01575000
1),AIRO(25),ECC(25),ALFA(25),BETA(25),GAMMA(25),EI(25),GAK(25) 01576000
2),TITLE(36)                                     01577000
DIMENSION CZ(25),CZ1(25),CZ2(25)               01578000
DIMENSION XKF(25),XCF(25),XKFF(25),XCFF(25)    01579000
DIMENSION QK(25),QC(25),QKP(25),QCP(25),QKHD(25),QCHD(25),QKF(25)01580000
1),QCF(25),QKPF(25),QCPF(25),QKHDF(25),QCHDF(25) 01581000
2),IB(12),K(25)                                  01582000
3),BKMX(25),BKMY(25),BCMX(25),BCMY(25),BCB(25)  01583000
4),BNB(25,6),BBB(25,6),BROB(25,7),BKB(25,6),BHB(25,6),BDB(25,6), 01584000
5BEB(25,6),RPM(50),INPRPM(50)                  01585000
COMMON/MAREAD/ TITLE,INPRPM, T,DT,FMAX, TMZ,FMZ1,FMZ2,FMZ301587000
1, TOLI, NOURPM, IASIGN,CRT,ACCEL                01588000
COMMON/MAFC/ EE,GG, EI,GAK                      01589000
COMMON/MAFUF1/DD,D,QL, NS,NB,IB                 01590000
COMMON/MAFU14/ FDOFIX                            01590500
COMMON/MAFU1/ TSTOP,K                            01591000
COMMON/MAFU2/ DN,AM,AID,AIRO,ECC,ALFA,BETA,GAMMA,GX,GY 01592000
1 /MAFU3/WHIVEL                                  01593000
COMMON/MAFU4/ CZ,CZ1,CZ2, XKF,XCF,XKFF,XCFF    01594000
COMMON/MAFU5/ QK,QC,QKP,QCP,QKHD,QCHD,QKF,QCF,QKPF,QCPF,QKHDF,QC01595000
1HDF, BKMX,BKMY,BCMX,BCMY,BCB                  01596000
COMMON/MAFU6/ BNB,BBB,BROB,BKB,BHB,BDB,BEB    01597000
COMMON/MAFU8/ TOLB                               01598000
COMMON/MAFU11/FD,FDOT                           01601000
X /MAFU13/Y(102)                                 01601100
X /SFP1/RPM,NPOINT,ICOND,CONTIN                 01602000
1 /SFP2/N1,NSM2,NSM3,NSP1,NS2,NS3,NS4, IDUM1(3) 01602100
2 /GARBG3/ICO,M,KI1,KI,I                        01602200
WRITE(6,471)                                     01602300
WRITE(6,400) TITLE                              01603000
399 FORMAT(10I10)                               01604000
400 FORMAT (1H0/ (20X,18A4))                    01605000
404 FORMAT (7X,1P6E13.4)                        01606000
490 FORMAT (14X,1P6E13.4)                       01607000
WRITE (6,401) NS, IASIGN, NOURPM, NPOINT, CRT, ACCEL 01608000
401 FORMAT (1H0/, 4X,8HNS =I3,14X, 'NUMBER OF ROTUR STATIONS (AL01609000
1LOWABLE RANGE:5=<NS=<25)'/4X,                01610000

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18HIASIGN =I3,14X, 70HA ROTOR STATION NUMBER AT WHICH THE WHIRL/SP01611000
2IN FREQUENCY RATIO WILL BE/34X,'PLOTTED ON CRT'/4X,8HNOORPM =I3,1401612000
3X,'THE NUMBER OF SPIN SPEEDS IN RPM AT OR NEAR WHICH 3-DIMENSIONAL01613000
1 ABSOLUTE ROTOR/34X'MODE SHAPE CRT GRAPHS ARE REQUIRED. THE SP101614000
2N SPEED RPM VALUES ARE LISTED/34X'UNDER INPRPM ARRAY.'/34X,'[ALLO01615000
2WABLE RANGE: 0=<NOORPM=<50)'/ 4X,8HNP0IN01616000
3T =I3, 01617000
614X,'THE NUMBER OF POINTS (ONE PER EACH INFEGRATION STEP) FOR EACH01618000
7 CRT GRAPH.'/34X,'(ALLOWABLE RANGE: 1=<NPOINT=<50)'/ 01619000
8 4X,8HCRT =I3,14X,31HCRT=0 MEANS CRT IS NOT REQUIRED01620000
8D/29X, 27HCRT=1 MEANS CRT IS REQUIRED/4X,8HACCEL =I3,14X,'ACCEL=01621000
90 MEANS A 3-DIMENSIONAL ROTOR MODE SHAPE CRT CORRESPONDING TO THAT01622000
9 AT'/34X, 'BEGINNING OF THE RUN WILL BE PROVIDED IF CONCURRENTLY C01623000
9RT=1.'/29X,'ACCEL=1 MEANS ONLY THE TRANSIENT-SPEED ROTOR MODE SHAPO1624000
9ES AT OR NEAR INPRPM/34X,'VALUES WILL BE PROVIDED IF CONCURRENTLY01625000
9 CRT=1.01625100
WRITE (6,453) 01626000
453 FORMAT(1H0, 3X,12HINPRPM ARRAY, 13X,'THE ROTOR SPIN SPEED RPM VAL01627000
1UES AT OR NEAR WHICH CRT GRAPHS FOR/29X,53H3-DIMENSIONAL ABSOLUTE01628000
2 ROTOR MODE SHAPES ARE REQUIRED) 01629000
WRITE (6,404) (INPRPM(I),I=1,NOORPM) 01630000
DT=DT/2. 01630500
WRITE (6,402) T, DT, TMAX, TOLI, TOLB,TSTOP 01631000
402 FORMAT ( 1H0/,4X,8HT =1PD13.4,4X, 23HINITIAL REAL TIME, SEC01632000
1./4X,8HDT =1PD13.4,4X'ESTIMATED INITIAL ' 01633000
2 'INTEGRATION STEP (REAL)TIME, SEC.'/4X8H01634000
2TMAX =1PD13.4,4X,31HTOTAL REAL TIME TO BE RUN, SEC./4X,8HTOLI 01635000
3=1PD13.4,4X, 31HINTEGRATION TOLERANCE, FRACTION/4X,8HTOLB =1PD1301636000
4.4,4X,54HTOLERANCE IN COMPUTING BEARING DISPLACEMENTS, FRACTION/4X01637000
5,8HTSTOP =1PD13.4,4X,'THE COMPUTER TIME ALLOWED FOR EACH SET OF D01637100
5ATA, MINUTES.01637200
WRITE (6,403) GX, GY, FMZ, FMZ1, FMZ2, TMZ3 01638000
403 FORMAT ( 1H0/,4X,8HGX =1PD13.4,4X, 46HGRAVITY OR G-LOADING IO1639000
1N X DIRECTION, IN/SEC**2/4X,8HGY =1PD13.4, 4X, 46HGRAVITY OR 01640000
2G-LOADING IN Y DIRECTION, IN/SEC**2/4X,8HTMZ =1PD13.4,4X, 47H01641000
3EXPONENT FOR SPEED SENSITIVE ROTOR DRIVE TORQUE/4X,8HTMZ1 =1PD1301642000
4.4,4X, 34HCOEFFICIENT FOR ROTOR DRIVE TORQUE/4X,8HTMZ2 =1PD13.401643000
5,4X, 52HCOEFFICIENT FOR ROTOR DRIVE TORQUE, (IN-LB-SEC)/RAD./4X01644000
6,8HTMZ3 =1PD13.4,4X, 43HCOEFFICIENT FOR ROTOR DRIVE TORQU01645000
7E, IN-LB ) 01646000
WRITE (6,405) 01647000
405 FORMAT (1H1/ 5X,'DD ARRAY',10X,'OUTSIDE DIAMETERS OF ROTOR SECTIO01648000

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	INS BETWEEN ADJACENT ROTOR STATIONS, IN.')	01649000
	WRITE (6,490) (DD(I),I=1,N1)	01651000
	WRITE (6,406)	01652000
406	FORMAT (1H0,4X,'D ARRAY',11X, 'INSIDE DIAMETERS OF ROTOR SECTION01653000	
	1S BETWEEN ADJACENT ROTOR STATIONS, IN.')	01654000
	WRITE (6,490) (D(I),I=1,N1)	01655000
	WRITE (6,407)	01656000
407	FORMAT (1H0,4X,'QL ARRAY',10X, 'ROTOR SECTION LENGTHS BETWEEN ADJ01657000	
	1ACENT ROTOR STATIONS, IN.')	01658000
	WRITE (6,490) (QL(I),I=1,N1)	01659000
	WRITE (6,408)	01660000
408	FORMAT (1H0,4X,'DN ARRAY',10X,'MATERIAL DENSITIES OF ROTOR SECTION01661000	
	1S BETWEEN ADJACENT STATIONS, LB/IN**3')	01662000
	WRITE (6,490) (DN(I),I=1,N1)	01663000
	WRITE (6,409)	01664000
409	FORMAT (1H0,4X,'EE ARRAY',10X, 'YOUNGS MODULI OF ROTOR SECTIONS 01665000	
	1BETWEEN ADJACENT STATIONS, LB/IN**2')	01666000
	WRITE (6,490) (EE(I),I=1,N1)	01667000
	WRITE (6,410)	01668000
410	FORMAT (1H0,4X,'GG ARRAY',10X, 'SHEAR MODULI OF ROTOR SECTIONS B01669000	
	1ETWEEN ADJACENT STATIONS, LB/IN**2')	01670000
	WRITE (6,490) (GG(I),I=1,N1)	01671000
	WRITE (6,418)	01672000
418	FURMAT (1H0,4X,'EI ARRAY',9X, 'DIRECT INPUT OF THE PRODUCTS OF Y001673000	
	1UNGS MODULI AND AREA MOMENTS OF INERTIA OF ROTOR '/23X,'SECTIONS BE01674000	
	2TWEEN ADJACENT STATIONS, LB-IN**2')	01675000
	WRITE (6,490) (EI(I),I=1,N1)	01676000
	WRITE (6,419)	01677000
419	FORMAT (1H0,4X,'GAK ARRAY',9X, 'DIRECT INPUT OF THE PRODUCTS OF SH01678000	
	1EAR MDDULI, CROSS-SECTIONAL AREAS '/23X'AND RECIPROCAL'S OF SHEAR ST01679000	
	2RESS CONCENTRATION FACTORS BETWEEN ADJACENT STATIONS, LB.')	01680000
	WRITE (6,490) (GAK(I),I=1,N1)	01681000
	WRITE (6,411)	01682000
411	FORMAT(1H0, 4X'AM ARRAY', 10X, 'ADDITIONAL ROTOR MASSES AT ROTOR 01683000	
	1STATIONS, (LB-SEC**2)/IN.')	01684000
	WRITE (6,404) (AM(I),I=1,NS)	01685000
	WRITE (6,412)	01686000
412	FORMAT (1H0,4X,'AID ARRAY', 9X, 'ADDITIONAL ROTOR TRANSVERSE MASS 01687000	
	1MOMENTS OF INERTIA AT ROTOR STATIONS, (LB-IN-SEC**2)')	01688000
	WRITE (6,404) (AID(I),I=1,NS)	01689000
	WRITE (6,413)	01690000
413	FORMAT (1H0,4X,'AIRO ARRAY',8X, 'ADDITIONAL ROTOR POLAR MASS MOMEN01691000	

	ITS OF INERTIA AT ROTOR STATIONS, (LB-IN-SEC**2)')	01692000
	WRITE (6,404) (AIRO(I),I=1,NS)	01693000
	WRITE (6,414)	01694000
414	FORMAT (1H0,4X,'ECC ARRAY', 9X, 'ROTOR MASS ECCENTRICITIES AT ROTOR STATIONS, IN.')'	01695000
	WRITE (6,404) (ECC(I),I=1,NS)	01696000
	WRITE (6,415)	01697000
415	FORMAT (1H0,4X,'ALFA ARRAY',8X, 'PHASE ANGLES FOR ROTOR MASS ECCENTRICITY VECTORS AT ROTOR STATIONS MEASURED FROM THE INITIAL ROTOR SPIN ANGULAR POSITION, DEGREES')'	01699000
	WRITE (6,404) (ALFA(I),I=1,NS)	01700000
	WRITE (6,416)	01701000
416	FORMAT (1H0//,5X,10HBETA ARRAY, 8X, 'INITIAL MISALIGNMENTS BETWEEN THE AXES OF THE MASS MOMENTS OF INERTIA AND THE ELASTIC AXES AT ROTOR STATIONS, DEGREES')'	01704000
	WRITE (6,404) (BETA(I),I=1,NS)	01705000
	WRITE (6,417)	01706000
417	FORMAT (1H0,4X,'GAMMA ARRAY',7X,'ANGULAR POSITIONS OF THE X-Y PLANE PROJECTIONS OF THE AXES OF MASS MOMENTS OF INERTIA AT ROTOR STATIONS MEASURED FROM THAT AT THE FIRST ROTOR STATION, DEGREES')'	01709000
	WRITE (6,404) (GAMMA(I),I=1,NS)	01710000
	WRITE (6,422)	01711000
422	FORMAT (1H0,4X,'CZ ARRAY', 10X,'PERSONAL FRICTION EXPONENTS AT ROTOR STATIONS, DIMENSIONLESS')'	01712000
	WRITE (6,404) (CZ(I),I=1,NS)	01713000
	WRITE (6,423)	01714000
423	FORMAT (1H0,4X,'CZ1 ARRAY',9X, 'TORSIONAL FRICTION COEFFICIENTS AT ROTOR STATIONS, DIMENSION OF CZ1*FDDT**CZ1/23X, IS IN-LB.')'	01715000
	WRITE (6,404) (CZ1(I),I=1,NS)	01716000
	WRITE (6,424)	01717000
424	FORMAT (1H0,4X,'CZ2 ARRAY',9X, 'TORSIONAL FRICTION COEFFICIENTS AT ROTOR STATIONS, (IN-LB-SEC)/RAD.')'	01718000
	WRITE (6,404) (CZ2(I),I=1,NS)	01719000
	WRITE (6,425)	01720000
425	FORMAT(1H0, 4X,9HXKF ARRAY,9X, 'WHIRL-FREQUENCY FACTORS FOR STIFFNESS FORCE COEFFICIENTS AT ROTOR STATIONS, '/23X, 'DIMENSIONLESS')'	01723000
	WRITE (6,404) (XKF(I),I=1,NS)	01724000
	WRITE (6,426)	01725000
426	FORMAT (1H0,4X,9HXCFC ARRAY, 9X, 'WHIRL-FREQUENCY FACTORS FOR DAMPING FORCE COEFFICIENTS AT ROTOR STATIONS, '/ 23X, 'DIMENSIONLESS')'	01726000
	WRITE (6,404) (XCFC(I),I=1,NS)	01727000
		01728000
		01729000
		01730000
		01731000
		01732000
		01733000


```

WRITE (6,437) 01776000
437 FORMAT (1H0,4X,10HQKPF ARRAY,8X,'OUT-OF-PHASE STIFFNESS MOMENT COEFFICIENTS AT ROTOR STATIONS, IN-LB.')
```

		01777000
		01778000
		01779000
		01780000

```

WRITE (6,404) (QKPF(I),I=1,NS)
438 FORMAT (1H0,4X,10HQCPF ARRAY,8X,'OUT-OF-PHASE DAMPING MOMENT COEFFICIENTS AT ROTOR STATIONS, IN-LB-SEC.')
```

		01781000
		01782000
		01783000
		01784000

```

WRITE (6,404) (QCPF(I),I=1,NS)
439 FORMAT (1H0,4X,'QKHDF ARRAY',7X,'OUT-OF-PHASE WHIRL AND SPIN VELOCITY SENSITIVE STIFFNESS MOMENT COEFFICIENTS AT /23X, 'ROTOR STATIONS, (LB-IN-SEC)/RAD.')
```

		01785000
		01786000
		01787000
		01788000
		01789000

```

WRITE (6,404) (QKHDF(I),I=1,NS)
440 FORMAT (1H0,4X,'QCHDF ARRAY',7X,'OUT-OF-PHASE WHIRL AND SPIN VELOCITY SENSITIVE DAMPING MOMENT COEFFICIENTS AT /23X, 'ROTOR STATIONS, (LB-IN-SEC**2)/RAD.')
```

		01790000
		01791000
		01792000
		01793000
		01794000

```

WRITE (6,461) NB
461 FORMAT (1H1/, 5X,12HNB = 13, 3X,'NUMBER OF NON-LINEAR STIFFNESS BEARINGS. (ALLOWABLE RANGE: 2=<NB=<12)')
```

		01795000
		01796000
		01797000

```

WRITE (6,462)
462 FORMAT( 1H0,4X8HIB ARRAY,10X,'ROTOR STATION NUMBERS FOR NON-LINEAR STIFFNESS BEARINGS')
```

		01799000
		01800000
		01801000

```

WRITE (6,399) (IB(I),I=1,NB)
463 FORMAT( 1H0,4X,'K ARRAY',11X, 'TOTAL NUMBER OF STIFFNESS SECTIONS FOR EACH OF THE NON-LINEAR STIFFNESS BEARINGS. /23X,'(ALLOWABLE RANGE: 1=<K=<6)')
```

		01802000
		01803000
		01804000
		01805000
		01806000

```

WRITE (6,399) (K(IB(I)),I=1,NB)
441 FORMAT( 1H0,4X,10HBKMX ARRAY,8X,63HNON-ISOTROPIC MOUNT STIFFNESS COEFFICIENTS IN X-DIRECTION FOR /23X, 'NON-LINEAR STIFFNESS BEARINGS, LB/IN.')
```

		01807000
		01808000
		01809000
		01810000
		01811000

```

WRITE (6,404) (BKMX(IB(I)),I=1,NB)
442 FORMAT( 1H0,4X,'BKMY ARRAY',8X, 63HNON-ISOTROPIC MOUNT STIFFNESS COEFFICIENTS IN Y-DIRECTION FOR /23X, 'NON-LINEAR STIFFNESS BEARINGS, LB/IN.')
```

		01812000
		01813000
		01814000
		01815000
		01816000

```

WRITE (6,442)
443 FORMAT( 1H0,4X'BCMCMX ARRAY',8X, 'NON-ISOTROPIC MOUNT DAMPING COEFFICIENTS AT ROTOR STATIONS, IN-LB-SEC.')
```

		01817000
--	--	----------

```

1 ICIENTS IN X-DIRECTION FOR '/23X, 'NON-LINEAR STIFFNESS BEARINGS, (LB01818000
1-SEC)/IN.' ) 01819000
WRITE (6,404) (BCM(IB(I)), I=1, NB) 01820000
WRITE (6,444) 01821000
444 FORMAT( 1H0,4X,10HBCMY ARRAY,8X,68HNON-ISOTROPIC MOUNT DAMPING CO01822000
EFFICIENTS IN Y-DIRECTION FOR /23X, 'NON-LINEAR STIFFNESS B01823000
2EARINGS, (LB-SEC)/IN.' ) 01824000
WRITE (6,404) (BCMY(IB(I)), I=1, NB) 01825000
WRITE (6,445) 01826000
445 FORMAT( 1H0,4X,9HBCB ARRAY,9X, 'BEARING DAMPING COEFFICIENTS FOR 01827000
1NON-LINEAR STIFFNESS BEARINGS, (LB-SEC)/IN.' ) 01828000
WRITE (6,404) (BCB(IB(I)), I=1, NB) 01829000
WRITE(6,420) 01830000
420 FORMAT(1H1//37X, 'NONLINEAR BEARING SPECIFICATIONS') 01830500
471 FORMAT(1H1) 01831000
DO 19 I=1, NB 01832000
KI=K(IB(I)) 01833000
WRITE (6,464) I 01834000
464 FORMAT(1H0,4X, 9HBNB ARRAY,9X, 'ROTOR SPIN-SPEED SENSITIVE BEARIN01835000
1G STIFFNESS COEFFICIENTS FOR K STIFFNESS'/23X, 'SECTIONS OF 01836000
2NON-LINEAR STIFFNESS BEARING NUMBER' I3) 01837000
19 WRITE (6,404) (BNB(IB(I),M), M=1, KI) 01838000
DO 20 I=1, NB 01839000
KI=K(IB(I)) 01840000
WRITE (6,465) I 01841000
465 FORMAT( 1H0,4X,9HBBB ARRAY,9X, 'ROTOR SPIN-SPEED SENSITIVE BEARIN01842000
1G STIFFNESS COEFFICIENTS FOR K STIFFNESS'/23X, 'SECTIONS OF 01843000
2BEARINGS FOR NON-LINEAR STIFFNESS BEARING NUMBER' I3) 01844000
20 WRITE (6,404) (BBB(IB(I),M), M=1, KI) 01845000
DO 13 I=1, NB 01846000
KI1=K(IB(I))+1 01847000
BROB(IB(I),1)=0.0 01848000
WRITE (6,466) I 01849000
466 FORMAT( 1H0, 4X, 'BROB ARRAY', 8X, 'UPPER BEARING-DISPLACEMENT LIMIT01850000
1S FOR K STIFFNESS SECTIONS OF BEARING '/23X, 'NUMBER' I3, ', IN.' ) 01851000
13 WRITE (6,404) (BROB(IB(I),M), M=2, KI1) 01852000
DO 14 I=1, NB 01853000
KI=K(IB(I)) 01854000
WRITE (6,467) I 01855000
467 FORMAT( 1H0,4X,9HBKB ARRAY,9X, 'NON-LINEAR BEARIN01856000
1G STIFFNESS COEFFICIENTS FOR K STIFFNESS' /23X, 'SECTIONS OF01857000
2 BEARING '/23X, 'NUMBER' I3) 01858000

```


480	WRITE(6,491)	01882500
491	FORMAT(1H0,30X,'RESTART ROTOR DEFLECTION AND VELOCITY ARRAY')	01882600
	WRITE(6,492)	01882700
492	FORMAT(1H0,4X,'ROTOR DEFLECTION VECTOR LENGTH, IN.')	01882800
	WRITE(6,404) (Y(I),I=1,NS)	01882900
	WRITE(6,493)	01883000
493	FORMAT(1H0,4X,'ROTOR DEFLECTION VECTOR ANGULAR POSITION, RAD.')	01883100
	NS2P1=NSP1+NS	01883300
	NS3P1=NS2P1+NS	01883400
	NS4P1=NS3P1+NS	01883500
	NS4P2=NS4P1+1	01883600
	WRITE(6,404) (Y(I),I=NSP1,NS2)	01884000
	WRITE(6,494)	01884100
494	FORMAT(1H0,4X,'ROTOR DEFLECTION VECTOR VELOCITIES')	01884200
	WRITE(6,404) (Y(I),I=NS2P1,NS3)	01884300
	WRITE(6,495)	01884400
495	FORMAT(1H0,4X,'ROTOR DEFLECTION VECTOR ANGULAR VELOCITIES')	01884500
	WRITE(6,404) (Y(I),I=NS3P1,NS4)	01884600
	WRITE(6,496) Y(NS4P1),Y(NS4P2)	01884700
496	FORMAT(1H0,4X,15HF = 1PD13.4,5X,'ROTOR SPIN ANGULAR DISO	01884800
	1PLACEMENT CUORDINATE, RAD. '/5X, 15HFDUT = 1PD13.4,5X,'ROTO	01884900
	2R SPIN VELOCIFY, RAD./SEC.')	01885000
500	WRITE(6,501)	01885010
501	FORMAT(/1H0,7X,'.....THE END OF INPUT DATA.....')	01885020
	RETURN	01885100
	END	01885200

Card Count 362


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SUBROUTINE STARUP                                00001000
  IMPLICIT REAL*8 (A-H,O-Z)                    00002000
  REAL Z,TITLE,INPRPM                          00004000
    DIMENSION  QL(25),DD2(25),D2(25),DD4(25),D4(25),QL2(25) 00005000
1,QLDNDD(25),DN(25),Q6LDND(25),DDPLD(25),DDL(25),Q1LDND(25),QM(25),00006000
2AM(25),QID(25),AID(25),QIRO(25),AIRO(25)      00007000
    DIMENSION Z(25), QME(25),                  ZS(25),SZ(25),ZQ00008000
1(25),RD(25)
    DIMENSION C(25,25),B(25,25)                00010000
    DIMENSION Y(102)                          00011000
    DIMENSION XX(25),YY(25),IB(12)            00013000
    DIMENSION AA(51,25),BB(51,25),           IA(51) 00015000
    DIMENSION  COSFAL(25),SINFAL(25),QMECOS(25),QM00016000
1ESIN(25)
    DIMENSION CMESIN(25,25),CMECOS(25,25)     00017000
    DIMENSION QIRFDD(25)                      00018000
    DIMENSION DD(25),D(25),ECC(25),ALFA(25), BETA(25),GAMMA(25) 00019000
    DIMENSION  QZ(25),QZOL(25),ZQOL(25),SZOL(25),ZSOL(25) 00022000
    DIMENSION  QK(25),QC(25),QKP(25),QCP(25),QKHD(25),QCHD(25),QKF(25) 00023000
1,QCF(25),QKPF(25),QCPF(25),QKHDF(25),QCHDF(25) 00024000
3,BKMX(25),BKMY(25),BCMX(25),BCMY(25),BCB(25),BKB(25,6),QKSAVE(25),00025000
4QCSAVE(25),BNB(25,6),BBB(25,6),BROB(25,7),BHB(25,6),BDB(25,6), 00026000
4BEB(25,6)
    DIMENSION  CZ(25),CZ1(25),CZ2(25),BRO(25,25),QRO(25) 00027000
    DIMENSION CQME(25,25),CONST(50),AS(50,50),PHAROO(25),BS(25,50) 00028000
    DIMENSION TITLE(36), INPRPM(50)          00029000
    COMMON/MAFUF1/DD,D,QL, NS,NB,IB          00030000
    COMMON/MAFUF2/ SZ,QZ,ZQ,SZOL,ZSOL,QZOL,ZQOL,QMLOV,QLL 00031000
    COMMON/MAFUF3/ IB1,IBNB                  00032000
    COMMON/MAFUF4/_Z                          00033000
    COMMON/MAFU14/ FDOFIX                     00034000
    COMMON/MAFU2/ DN,AM,AID,AIRO,ECC,ALFA,BETA,GAMMA 00035000
    COMMON/MAFU4/ CZ,CZ1,CZ2                  00036000
    COMMON/MAFU5/ QK,QC,QKP,QCP,QKHD,QCHD,QKF,QCF,QKPF,QCPF,QKHDF,QC00037000
1HDF,BKMX,BKMY,BCMX,BCMY,BCB
    COMMON/MAFU6/BNB,BBB,BROB,BKB,BHB,BDB,BEB 00038000
    COMMON/MAFU8/ TOLB                        00039000
    COMMON/MAFU9/ C,B                         00040000
    COMMON/MAFU10/ RD,PHAROO                  00042000
    COMMON/MAFU11/ FD,FDOT                    00043000
    COMMON/MAIFU/ Q,S                         00044000
    COMMON/MAREAD/ TITLE,INPRPM,             TTT,DT,TMAX, TMZ, TMZ1, TMZ2, TMZ300047000

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```

1,      TQLI
COMMON/MAFU13/Y.
FDSAVE=FD
FOSAVE=FDOT
PI= 3.14159265358979324
G=386.088
U=4./3.
V=PI/180.
W=PI/(128.*G)
E=PI/(8.*G)
F=FD*V
FDOT=F*DOT*V*6.
DO 90 I=1,NB
IBI=IB(I)
QKSAVE(IBI)=QK(IBI)
QCSAVE(IBI)=QC(IBI)
QK(IBI)=QK(IBI)+1/(2/(BKM(X)IBI)+BKM(Y)IBI))+1/(
(1)+BNB(IBI,1)*(FDOT-FDOFIX))) (BKB(IBI,
IF(BCB(IBI).EQ.0.)BCB(IBI)=1.D-16
IF(BCMX(IBI).EQ.0.)BCMX(IBI)=1.D-16
IF(BCMY(IBI).EQ.0.)BCMY(IBI)=1.D-16
QC(IBI)=1./(1./BCB(IBI)+2./(BCMX(IBI)+BCMY(IBI)))+QC(IBI)
90 QC(IBI)=0.
DO 50 I=1,NS.
ZS(I)=-SZ(I)
DD2(I)= DD(I)**2
D2 (I)= D(I)**2
DD4(I)=DD2(I)**2
D4(I) =D2(I)**2
QL2(I)= QL(I)**2.
QLDND(I)=QL(I)* DN(I)*(DD2(I)-D2(I))
Q6LDND(I) = W*QLDND(I)
DDPLD(I) = DD2(I)+D2(I)
DDL(I) = DDPLD(I)+ U*QL2(I)
50 Q1LDND(I) = E*QLDND(I)
QM(1)=Q1LDND(1) + AM(1)
QM(NS)=Q1LDND(NS-1) +AM(NS)
QID(1) = Q6LDND(1)*DDL(1) + AID(1)
QID(NS)= Q6LDND(NS-1)*DDL(NS-1) +AID(NS)
QIRO(1)=2.*Q6LDND(1)*DDPLD(1)+AIRO(1)
QIRO(NS)=2.*Q6LDND(NS-1)*DDPLD(NS-1) +AIRO(NS)
NSM1=NS-1
00048000
00049000
00051000
00052000
00059000
00060000
00061000
00062000
00063000
00064000
00065000
00066000
00053000
00054000
00055000
00056000
00057000
00057500
00058000
00058100
00058200
00058300
00058400
00067000
00068000
00069000
00070000
00071000
00072000
00073000
00074000
00075000
00076000
00077000
00078000
00079000
00080000
00081000
00082000
00083000
00084000
00085000

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DO 55 I=2,NSM1                                00086000
QM(I) =Q1LDND(I-1)+Q1LDND(I) +      AM(I)    00087000
QID(I)=Q6LDND(I-1)*DDL(I-1)+Q6LDND(I)*DDL(I)+AID(I) 00088000
55 QIRO(I) = 2.*(Q6LDND(I-1)*DDPLD(I-1)+Q6LDND(I)*DDPLD(I)) +AIRU(I) 00089000
DO 103 I=1,NS                                  00100000
103 QME(I)=QM(I)*ECC(I)                      00101000
WRITE(6, 80)                                   00103000
80 FORMAT (1H1//14X,28HFORCE INFLUENCE COEFFICIENTS) 00104000
DO 81 I=1,NS                                  00105000
81 WRITE(6,592) I, (C(I,J),J=1,NS)           00106000
592 FORMAT(1H04X3HROWI3/(1P7D15.6))         00107000
WRITE (6,82)                                   00108000
82 FORMAT (1H1//14X,29HMOMENT INFLUENCE COEFFICIENTS) 00109000
DO 83 I=1,NS                                  00110000
83 WRITE(6,592) I, (B(I,J),J=1,NS)           00111000
DO 45 J=1,NS                                  00112000
AA(J,1)=-C(1,J)*QM(1)+QID(1)*B(1,J)/QL(1)+QID(2)*B(2,J)/(QL(2) 00113000
+QL(1))
1 AA(J,2)=-C(2,J)*QM(2)-QID(1)*B(1,J)/QL(1)+QID(3)*B(3,J)/(QL(3)+00114000
1QL(2))
NSM2=NS-2                                     00116000
NSM3=NS-3                                     00117000
AA(J,NSM1)=-C(NSM1,J)*QM(NSM1)-QID(NSM2)*B(NSM2,J)/(QL(NSM2)+QL 00118000
1(NSM3))+QID(NS)*B(NS,J)/QL(NSM1)           00119000
45 AA(J,NS)=-C(NS,J)*QM(NS)-QID(NSM1)*B(NSM1,J)/(QL(NSM1)+QL(NSM2) 00120000
1)-QID(NS)*B(NS,J)/QL(NSM1)                 00121000
DO 47 J=1,NS                                  00122000
DO 47 I=3,NSM2                                00123000
47 AA(J,I)=-C(I,J)*QM(I)-QID(I-1)*B(I-1,J)/(QL(I-1)+QL(I-2))+QID(I 00124000
1+1)*B(I+1,J)/(QL(I+1)+QL(I))              00125000
AA(IB1,1) = -(Q-Z(1))*QM(1)-QID(1)/QL(1)-QID(2)/(QL(2)+QL(1)) 00126000
AA(IB1,2) = -(Q-Z(2))*QM(2)+QID(1)/QL(1)-QID(3)/(QL(3)+QL(2)) 00127000
AA(IB1,NSM1)=- (Q-Z(NSM1))*QM(NSM1)+QID(NSM2)/(QL(NSM2)+QL(NSM3) 00128000
1)-QID(NS)/QL(NSM1)                          00129000
AA(IB1,NS) = -(Q-Z(NS))*QM(NS)+QID(NSM1)/(QL(NSM1)+QL(NSM2))+QID 00130000
1D(NS)/QL(NSM1)                              00131000
DO 49 I=3,NSM2                                00132000
49 AA(IB1,I)=- (Q-Z(I))*QM(I)+QID(I-1)/(QL(I-1)+QL(I-2))-QID(I+1)/ 00133000
1QL(I+1)+QL(I))                             00134000
AA(IBNB,1) = -(Z(1)-S)*QM(1)+QID(1)/QL(1)+QID(2)/(QL(2)+QL(1)) 00135000
AA(IBNB,2) = -(Z(2)-S)*QM(2)-QID(1)/QL(1)+QID(3)/(QL(3)+QL(2)) 00136000
AA(IBNB,NSM1)=- (Z(NSM1)-S)*QM(NSM1)-QID(NSM2)/(QL(NSM2)+QL(NSM3) 00137000
00138000

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	1)+QID(NS)/QL(NSM1)	00139000
	AA(IBNB,NS) = -(Z(NS)-S)*QM(NS)-QID(NSM1)/(QL(NSM1)+QL(NSM2))-QI	00140000
	1D(NS)/QL(NSM1)	00141000
	DO 48 I=3,NSM2	00142000
48	AA(IBNB,I)=- (Z(I)-S)*QM(I)-QID(I-1)/(QL(I-1)+QL(I-2))+QID(I+1)/	00143000
	1(QL(I+1)+QL(I))	00144000
	NSP1=NS+1	00145000
	NS2 =2*NS	00147000
	DO 53 K=NSP1,NS2	00148000
	J=K-NS	00149000
	BB(K,1)=-C(1,J)*QM(1)+QID(1)*B(1,J)/QL(1)+QID(2)*B(2,J)/(QL(2)+	00150000
	1QL(1))	00151000
	BB(K,2)=-C(2,J)*QM(2)-QID(1)*B(1,J)/QL(1)+QID(3)*B(3,J)/(QL(3)+	00152000
	1QL(2))	00153000
	BB(K,NSM1)=-C(NSM1,J)*QM(NSM1)-QID(NSM2)*B(NSM2,J)/(QL(NSM2)+QL	00154000
	1(NSM3))+QID(NS)*B(NS,J)/QL(NSM1)	00155000
53	BB(K,NS)=-C(NS,J)*QM(NS)-QID(NSM1)*B(NSM1,J)/(QL(NSM1)+QL(NSM2)	00156000
	1)-QID(NS)*B(NS,J)/QL(NSM1)	00157000
	DO 65 K=NSP1,NS2	00158000
	J=K-NS	00159000
	DO 65 I=3,NSM2	00160000
65	BB(K,I)=-C(I,J)*QM(I)-QID(I-1)*B(I-1,J)/(QL(I-1)+QL(I-2))+QID(I	00161000
	1+1)*B(I+1,J)/(QL(I+1)+QL(I))	00162000
	NSB1=NS+IB1	00163000
	BB(NSB1,1)=- (Q-Z(1))*QM(1)-QID(1)/QL(1)-QID(2)/(QL(2)+QL(1))	00164000
	BB(NSB1,2)=- (Q-Z(2))*QM(2)+QID(1)/QL(1)-QID(3)/(QL(3)+QL(2))	00165000
	BB(NSB1,NSM1)=- (Q-Z(NSM1))*QM(NSM1)+QID(NSM2)/(QL(NSM2)+QL(NSM	00166000
	13))-QID(NS)/QL(NSM1)	00167000
	BB(NSB1,NS)=- (Q-Z(NS))*QM(NS)+QID(NSM1)/(QL(NSM1)+QL(NSM2))+QID	00168000
	1(NS)/QL(NSM1)	00169000
	DO 66 I=3,NSM2	00170000
66	BB(NSB1,I)=- (Q-Z(I))*QM(I)+QID(I-1)/(QL(I-1)+QL(I-2))-QID(I+1)/	00171000
	1(QL(I+1)+QL(I))	00172000
	NSNB=NS+IBNB	00173000
	BB(NSNB,1)=- (Z(1)-S)*QM(1)+QID(1)/QL(1)+QID(2)/(QL(2)+QL(1))	00174000
	BB(NSNB,2)=- (Z(2)-S)*QM(2)-QID(1)/QL(1)+QID(3)/(QL(3)+QL(2))	00175000
	BB(NSNB,NSM1)=- (Z(NSM1)-S)*QM(NSM1)-QID(NSM2)/(QL(NSM2)+QL(NSM	00176000
	13))+QID(NS)/QL(NSM1)	00177000
	BB(NSNB,NS)=- (Z(NS)-S)*QM(NS)-QID(NSM1)/(QL(NSM1)+QL(NSM2))-QID	00178000
	1(NS)/QL(NSM1)	00179000
	DO 56 I=3,NSM2	00180000
56	BB(NSNB,I)=- (Z(I)-S)*QM(I)-QID(I-1)/(QL(I-1)+QL(I-2))+QID(I+1)/	00181000

I(QL(I+1)+QL(I))	00182000
DO 67 J=1,NS	00183000
DO 67 I=1,NS	00184000
QME(I)=QM(I)*ECC(I)	00185000
CQME(I,J)=C(I,J)*QME(I)	00186000
FALFA = F+ ALFA(I)*V	00187000
SINFAL(I)=DSIN(FALFA)	00188000
QMESIN(I)=QME(I)*SINFAL(I)	00189000
COSFAL(I)=DCOS(FALFA)	00190000
QMECOS(I)=QME(I)*COSFAL(I)	00191000
CMESIN(I,J)=CQME(I,J)*SINFAL(I)	00192000
67 CMECOS(I,J)=CQME(I,J)*COSFAL(I)	00193000
DO 71 I=1,NS	00194000
71 QIRFDO(I)=QIRO(I)*FDOT	00195000
NS2 =2*NS	00196000
FDOTSQ=FDOT**2	00198000
FF=0.0	00199000
DO 690 J=1,NS2	00200000
DO 690 I=1,NS2	00201000
690 AS(J,I)=0	00202000
DO 700 J=1,NS	00205000
BRO(J,1)= B(1,J)*QIRO(1)/QL(1) +B(2,J)*QIRO(2)/(QL(1)+QL(2))	00206000
BRO(J,2)=-B(1,J)*QIRO(1)/QL(1) +B(3,J)*QIRO(3)/(QL(2)+QL(3))	00207000
BRO(J,NSM1)=-B(NSM2,J)*QIRO(NSM2)/(QL(NSM3)+QL(NSM2))+ B(NS,J)*	00208000
QIRO(NS)/QL(NSM1)	00209000
700 BRO(J,NS)=-B(NSM1,J)*QIRO(NSM1)/(QL(NSM2)+QL(NSM1)) -B(NS,J)*QI	00210000
RO(NS)/QL(NSM1)	00211000
DO 710 J=1,NS	00212000
DO 710 I=3,NSM2	00213000
710 BRO(J,I)=-B(I-1,J)*QIRO(I-1)/(QL(I-2)+QL(I-1))+ B(I+1,J)*QIRO	00214000
(I+1)/(QL(I)+QL(I+1))	00215000
QRO(1)= QIRO(1)/QL(1)+ QIRO(2)/(QL(1)+QL(2))	00216000
QRO(2)= -QIRO(1)/QL(1) +QIRO(3)/(QL(2)+QL(3))	00217000
QRO(NSM1)= -QIRO(NSM2)/(QL(NSM3)+QL(NSM2)) +QIRO(NS)/QL(NSM1)	00218000
QRO(NS)=-QIRO(NSM1)/(QL(NSM2)+QL(NSM1)) -QIRO(NS)/QL(NSM1)	00219000
DO 720 I=3,NSM2	00220000
720 QRO(I)=-QIRO(I-1)/(QL(I-2)+QL(I-1)) +QIRO(I+1)/(QL(I)+QL(I+1))	00221000
1)	00222000
DO 730 J=1,NS	00223000
CONST(J)=0	00224000
DO 740 I=1,NS	00225000
740 CONST(J)=CONST(J)-CMECOS(I,J)	00226000

730	CONST(J)= FDOTSQ*CONST(J)	00227000
	CONST(IB1)=0	00228000
	CONST(IBNB)=0	00229000
	DO 750 I=1,NS	00230000
	CONST(IB1)=CONST(IB1) -QZ(I)* QMECUS(I)	00231000
750	CONST(IBNB)=CONST(IBNB) -ZS(I) *QMECOS(I)	00232000
	CONST(IB1)=FDOTSQ*CONST(IB1)	00233000
	CONST(IBNB)= FDOTSQ*CONST(IBNB)	00234000
	DO 760 J=NSP1,NS2	00235000
	K=J-NS	00236000
	CONST(J)=0	00237000
	DU 770 I=1,NS	00238000
770	CONST(J)= CONST(J)-CMESIN(I,K)	00239000
760	CONST(J)= FDOTSQ*CONST(J)	00240000
	CONST(NSB1)=0	00241000
	CONST(NSNB)=0	00242000
	DO 775 I=1,NS	00243000
	CONST(NSB1)= CONST(NSB1) -QZ(I)*QMESIN(I)	00244000
775	CONST(NSNB)= CONST(NSNB) -ZS(I)*QMESIN(I)	00245000
	CONST(NSB1)= FDOTSQ* CONST(NSB1)	00246000
	CONST(NSNB)= FDOTSQ* CONST(NSNB)	00247000
	DO 790 J=1,NS	00248000
	DO 790 I=1,NS	00249000
	AS(J,I) =-C(I,J)*QK(I) +FDOTSQ*BRO(J,I)	20052000
790	BS(J,I)=AS(J,I)	20052500
	DO 800 J=1,NS	00251000
	DO 800 I=NSP1,NS2	00252000
	M=I-NS	00253000
	AS(J,I)= C(M,J)*FDOT*QC(M)	20055000
800	BS(J,I)=-AS(J,I)	20055500
	DO 810 J=1,NS	00255000
	AS(J,IB1)= AS(J,IB1)+1-ZS(J)/QLL	00256000
	BS(J,IB1)=AS(J,IB1)	20057500
	AS(J,IBNB)=AS(J,IBNB)+ZS(J)/QLL	00257000
	BS(J,IBNB)=AS(J,IBNB)	20058500
	AS(J,J) = AS(J,J)-1	20059000
810	BS(J,J)=AS(J,J)	20059500
	DO 820 I=1,NS	00259000
	AS(IB1,I)=-QZ(I)*QK(I)- FDOTSQ*QRO(I)	20065000
820	BS(IB1,I)=AS(IB1,I)	20065500
	DO 830 I=NSP1,NS2	00261000
	M=I-NS	00262000

173

	AS(IB1,I)= QZ(M)*FDOT*QC(M)	20068000
830	BS(IB1,I)=-AS(IB1,I)	20068500
	DO 840 I=1,NS	00264000
	AS(IBNB,I)= -ZS(I)*QK(I) +FDOTSQ* QRD(I)	20075000
840	BS(IBNB,I)=AS(IBNB,I)	20075500
	DO .850 I=NSP1,NS2	00266000
	M=I-NS	00267000
	AS(IBNB,I)= ZS(M)*QC(M)*FDOT	20077000
850	BS(IBNB,I)=-AS(IBNB,I)	20077500
	DO 860 J=1,NS	00271000
	DO 860 I=1,NS	00272000
860	AS(J,I)= AS(J,I) -FDOTSQ*AA(J,I)	00273000
	DO 950 J=1,NS	00297000
	DO 950 I=1,NS	00298000
	K=J+NS	00299000
950	BS(J,I)= BS(J,I) -FDOTSQ*BB(K,I)	00300000
	DO 630 J=NSP1,NS2	00301000
	M=J-NS	00302000
	DO 630 I=NSP1,NS2	00303000
	K=I-NS	00304000
	AS(J,I)=BS(M,K)	00305000
630	AS(J,K)=BS(M,I)	00306000
171	KL=ISIMDD(50,NS2,1,AS,CONST,FF,IA)	00321000
	DO 620 I=1,NS	00322000
	IPNS=I+NS	00323000
	XX(I)=AS(I,1)	00324000
620	YY(I)=AS(IPNS,1)	00325000
	WRITE(6,471)	00325500
	WRITE(6,600)	00326000
600	FORMAT(1H0,4X,'COMPUTED STARTING XX ARRAY')	00327000
	WRITE(6,404) (XX(I),I=1,NS)	00328000
	WRITE(6,610)	00329000
610	FORMAT(1H0,4X,'COMPUTED STARTING YY ARRAY')	00330000
	WRITE(6,404) (YY(I),I=1,NS)	00331000
	DO 870 I=1,NS	00332000
	RO(I)=DSQRT(XX(I)**2+YY(I)**2)	00333000
	PHAROO(I)=DATN2D(YY(I),XX(I))	00334000
870	IF(PHAROO(I).LT.0) PHAROO(I)=360+PHAROO(I)	00335000
404	FORMAT (7X,1P6E13.4)	00336000
471	FORMAT(1H1)	00337000
	WRITE. (6,447)	00338000
447	FORMAT(1H0,4X,	8HR0 00339000

IARRAY,10X,'INITIAL DISPLACEMENT VECTORS (WHICH CAN NOT BE ZERO) FROM	00340000
10M THEIR RESPECTIVE ZERO LOAD'/23X,'POSITIONS, IN.'	00341000
WRITE (6,404) (RO(I),I=1,NS)	00342000
WRITE (6,449)	00343000
449 FORMAT(1H0,4X'PHAROD ARRAY',6X'INITIAL PHASE ANGLES FOR THE DISPLA	00344000
ICEMENT VECTORS, DEGREES.'/23X,'THE PHASE ANGLES CAN NOT BE ZERO DR	00345000
2 MULTIPLES OF 90 DEGREES.'	00346000
WRITE (6,404) (PHAROD(I),I=1,NS)	00347000
WRITE(6,471)	00350000
IF (KL.NE.3) GO TO 560	00351000
WRITE (6,540) KL	00352000
540 FORMAT(1H1/10X,4HKL =I1)	00353000
CALL EXIT	00354000
560 FD=FDSAVE	00355000
FDOT=FOSAVE	00356000
DO 980 I=1,NB	00357000
IBI=IB(I)	00358000
QK(IBI)=QKSAVE(IBI)	00359000
980 QC(IBI)=QCSAVE(IBI)	00360000
RETURN	00361000
END	00362000

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Card Count 315


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SUBROUTINE FUND(N,T,Y,BD)
IMPLICIT REAL*8 (A-H,O-Z)
REAL Z,TITLE,INPRPM
DIMENSION QL(25),DN(25),
1QM(25),Z(25),QME(25),SZ(25),ZQ(25),QMZ(25),ROSQ(25), Y(102),
2AM(25),QID(25),AID(25),QIRO(25),AIRO(25)
3,DELTX(25),DELTY(25),COSAB(25),SINAB(25),XDOT(25),YDOT(25),XX(25),
4YY(25),IB(12),XBDOT(25),YBDOT(25),BCB(25),BCMX(25),BCMY(25),KK(25)
5,C(25,25),B(25,25),BKMX(25),BKMY(25)
DIMENSION BNB(25,6),BBB(25,6),BROB(25,7),BKB(25,6),BHB(25,6),BDB
1 (25,6),BEB(25,6)
DIMENSION AA(51,25),BB(51,25),AROT(51,51),IA(51)
1,CC(51),DRU(51),DSTA(51)
DIMENSION COSFAL(25),SINFAL(25),QMECOS(25),QM
1ESIN(25),QIDBSN(25),QIDBCS(25),QMEFSQ(25),QKHDF(25),QCHDF(25),
2IRFDD(25),QKHDF(25),QCHDFF(25),BRGROP(25),XB(25),YB(25),AMX(25),
3AMY(25),CZ(25),CZ1(25),CZ2(25),COSFGA(25),SINFGA(25)
DIMENSION AFX(25),AFY(25),AMX1(25),AMY1(25),AMX2(25),AMY2(25)
DIMENSION BD(102),WHRVLO(25),QKHDD(25),QCHDD(25)
DIMENSION DD(25),D(25),ECC(25),ALFA(25),BETA(25),GAMMA(25),
1 A(25)
DIMENSION QZ(25),QZOL(25),ZQOL(25),SZOL(25),ZSOL(25)
DIMENSION XKF(25),XCF(25),XKFF(25),XCF(25), QK(25),QC(25),QKP
1(25),QCP(25),QKHD(25),QCHD(25),QKF(25),QKF(25),QKPF(25),QCPF(25)
DIMENSION TITLE(36),INPRPM(50)
COMMON/MAFUF1/DD,D,QL, NS,NB,IB
COMMON/MAFUF2/ SZ,QZ,ZQ,SZOL,ZSOL,QZOL,ZQOL,QMLOV,QLL
COMMON/MAFUF3/ IB1,IBNB
COMMON/MAFUF4/ Z
COMMON/MAFU14/ FDOFIX
COMMON/MAFU0/ ITIM,INT, KK
COMMON/MAFU2/ DN,AM,AID,AIRO,ECC,ALFA,BETA,GAMMA,GX,GY
COMMON/MAFU4/ CZ,CZ1,CZ2, XKF,XCF,XKFF,XCF
COMMON/MAFU5/ QK,QC,QKP,QCP,QKHD,QCHD,QKF,QCF,QKPF,QCPF,QKHDF,QC
1HDF, BKMX,BKMY,BCMX,BCMY,BCB
COMMON/MAFU6/ BNB,BBB,BROB,BKB,BHB,BDB,BEB
COMMON /MAFU7/ROSQ, WHRVLO,DELTX,DELTY
COMMON/MAFU8/ TOLB
COMMON/MAFU9/ C,B
X /MAFU12/QM,QID,QIRO
1 /MAFU11/FD,FDDT
COMMON/MAIFU/ Q,S

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00102000
00103000
00105000
00106000
00107000
00108000
00109000
00110000
00111000
00123000
00124000
00125000
0000126000
000127000
00131000
00131100
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00140000
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00145000
00146000
00147000
00147600
00148000
00149000
00150000
000151000
00152000
00153000
00154000
00155000
00156000
00157000
00157100
00158000

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COMMON/MAREAD/ TITLE,INPRPM,          TTT,DT,TMAX,  TMZ,TMZ1,TMZ2,TMZ300159000
1,          TOLI                               00160000
2          /SFP2/NSM1,NSM2,NSM3,NSP1,NS2,NS3,NS4,NS4P1,NS4P2,NS2P1 00160010
3          /GARBG5/QME                          00160020
4          /GARBG6/QMZ,AA,BB,AROT,CC,DRD,DSTA,  00160030
5CDSAB,SINAB,QIRFDD,QKHDFE,QCHDFE,BRGROP,AFX,AFY,AMX1,AMX2,AMY1,  00160040
6AMY2,QCHDD,QKHDD,A,AMX,AMY,COSFGA,SINFGA,    00160050
7TRIG,COSFAL,SINFAL,QMECOS,QMESIN,QIDBET,QIDBSN,QIDBCS,QMEFSQ,  00160060
8IA                                             00160070
9          /GARBG7/XX,YY,XB,YB,XDOT,YDOT,XBDOT,YBDOT 00160080
  IF (INT .EQ.2) GO TO 406                      00161000
    PI= 3.14159265358979324                    00162000
    V=PI/180.                                   00165000
  DO 103 I=1,NS                                00200000
103 QME(I)=QM(I)*ECC(I)                        00201000
    INT=2                                        00203000
406 DO 2 I=1,N                                  00203000
  IF(Y(I).GE.1.D14) GO TO 3                    00204000
  2 CONTINUE                                    00205000
  GO TO 4                                        00206000
  3 RETURN                                      00207000
  4 FDOT=Y(NS4P2)                               00210000
    DO 79 I=1,NS                                00211000
      INS=I+NS                                  00212000
      I2NS=I+NS2                                00213000
      I3NS=I+NS3                                00214000
      COSAB(I)=DCOS(Y(INS))                     00215000
      SINAB(I)=DSIN(Y(INS))                     00216000
      XNUTR=Y(I)*COSAB(I)                       00217000
      YNUTR=Y(I)*SINAB(I)                       00218000
      XX(I)=XNUTR-DELTX(I)                      00219000
      YY(I)=YNUTR-DELTY(I)                      00220000
      XDOT(I)=Y(I2NS)*COSAB(I)-Y(I)*Y(I3NS)*SINAB(I) 00221000
      YDOT(I)=Y(I2NS)*SINAB(I)+Y(I)*Y(I3NS)*COSAB(I) 00222000
79  WHRVLO(I)=Y(I3NS)                           00223000
80 CALL BRGXY                                    00223100
    DO 45 J=1,NS                                 00312000
      AA(J,1)=-C(1,J)*QM(1)+QID(1)*B(1,J)/QL(1)+QID(2)*B(2,J)/(QL(2)+ 00313000
      +QL(1))                                     00314000
      AA(J,2)=-C(2,J)*QM(2)-QID(1)*B(1,J)/QL(1)+QID(3)*B(3,J)/(QL(3)+ 00315000
      +QL(2))                                     00316000
100  AA(J,NSM1)=-C(NSM1,J)*QM(NSM1)-QID(NSM2)*B(NSM2,J)/(QL(NSM2)+QL 00319000
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1(NSM3))+QID(NS)*B(NS,J)/QL(NSM1) 00320000
45 AA(J,NS)=-C(NS,J)*QM(NS)-QID(NSM1)*B(NSM1,J)/(QL(NSM1)+QL(NSM2))00321000
1)-QID(NS)*B(NS,J)/QL(NSM1) 00322000
DO 47 J=1,NS 00323000
DO 47 I=3,NSM2 00324000
47 AA(J,I)=-C(I,J)*QM(I)-QID(I-1)*B(I-1,J)/(QL(I-1)+QL(I-2))+QID(I00325000
1+1)*B(I+1,J)/(QL(I+1)+QL(I)) 00326000
AA(IB1,1) = -(Q-Z(1))*QM(1)-QID(1)/QL(1)-QID(2)/(QL(2)+QL(1)) 00327000
AA(IB1,2) = -(Q-Z(2))*QM(2)+QID(1)/QL(1)-QID(3)/(QL(3)+QL(2)) 00328000
AA(IB1,NSM1)=- (Q-Z(NSM1))*QM(NSM1)+QID(NSM2)/(QL(NSM2)+QL(NSM3))00329000
1)-QID(NS)/QL(NSM1) 00330000
AA(IB1,NS) = -(Q-Z(NS))*QM(NS)+QID(NSM1)/(QL(NSM1)+QL(NSM2))+QI00331000
1D(NS)/QL(NSM1) 00332000
DO 49 I=3,NSM2 00333000
49 AA(IB1,I)=- (Q-Z(I))*QM(I)+QID(I-1)/(QL(I-1)+QL(I-2))-QID(I+1)/00334000
1(QL(I+1)+QL(I)) 00335000
AA(IBNB,1) = -(Z(1)-S)*QM(1)+QID(1)/QL(1)+QID(2)/(QL(2)+QL(1)) 00336000
AA(IBNB,2) = -(Z(2)-S)*QM(2)-QID(1)/QL(1)+QID(3)/(QL(3)+QL(2)) 00337000
AA(IBNB,NSM1)=- (Z(NSM1)-S)*QM(NSM1)-QID(NSM2)/(QL(NSM2)+QL(NSM3))00338000
1)+QID(NS)/QL(NSM1) 00339000
AA(IBNB,NS) = -(Z(NS)-S)*QM(NS)-QID(NSM1)/(QL(NSM1)+QL(NSM2))-QI00340000
1D(NS)/QL(NSM1) 00341000
DO 48 I=3,NSM2 00342000
48 AA(IBNB,I)=- (Z(I)-S)*QM(I)-QID(I-1)/(QL(I-1)+QL(I-2))+QID(I+1)/00343000
1(QL(I+1)+QL(I)) 00344000
DO 53 K=NSP1,NS2 00348000
J=K-NS 00349000
BB(K,1)=-C(1,J)*QM(1)+QID(1)*B(1,J)/QL(1)+QID(2)*B(2,J)/(QL(2)+00350000
1QL(1)) 00351000
BB(K,2)=-C(2,J)*QM(2)-QID(1)*B(1,J)/QL(1)+QID(3)*B(3,J)/(QL(3)+00352000
1QL(2)) 00353000
BB(K,NSM1)=-C(NSM1,J)*QM(NSM1)-QID(NSM2)*B(NSM2,J)/(QL(NSM2)+QL00354000
1(NSM3))+QID(NS)*B(NS,J)/QL(NSM1) 00355000
53 BB(K,NS)=-C(NS,J)*QM(NS)-QID(NSM1)*B(NSM1,J)/(QL(NSM1)+QL(NSM2))00356000
1)-QID(NS)*B(NS,J)/QL(NSM1) 00357000
DO 65 K=NSP1,NS2 00358000
J=K-NS 00359000
DO 65 I=3,NSM2 00360000
65 BB(K,I)=-C(I,J)*QM(I)-QID(I-1)*B(I-1,J)/(QL(I-1)+QL(I-2))+QID(I00361000
1+1)*B(I+1,J)/(QL(I+1)+QL(I)) 00362000
NSB1=NS+IB1 00363000
BB(NSB1,1)=- (Q-Z(1))*QM(1)-QID(1)/QL(1)-QID(2)/(QL(2)+QL(1)) 00364000

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BB(NSB1,2)=- (Q-Z(2))*QM(2)+QID(1)/QL(1)-QID(3)/(QL(3)+QL(2)) 00365000
BB(NSB1,NSM1)=- (Q-Z(NSM1))*QM(NSM1)+QID(NSM2)/(QL(NSM2)+QL(NSM00366000
13))-QID(NS)/QL(NSM1) 00367000
BB(NSB1,NS)=- (Q-Z(NS))*QM(NS)+QID(NSM1)/(QL(NSM1)+QL(NSM2))+QID00368000
1(NS)/QL(NSM1) 00369000
DU 66 I=3,NSM2 00370000
66 BB(NSB1,I)=- (Q-Z(I))*QM(I)+QID(I-1)/(QL(I-1)+QL(I-2))-QID(I+1)/00371000
1(QL(I+1)+QL(I)) 00372000
NSNB=NS+IBNB 00373000
BB(NSNB,1)=- (Z(1)-S)*QM(1)+QID(1)/QL(1)+QID(2)/(QL(2)+QL(1)) 00374000
BB(NSNB,2)=- (Z(2)-S)*QM(2)-QID(1)/QL(1)+QID(3)/(QL(3)+QL(2)) 00375000
BB(NSNB,NSM1)=- (Z(NSM1)-S)*QM(NSM1)-QID(NSM2)/(QL(NSM2)+QL(NSM00376000
13))+QID(NS)/QL(NSM1) 00377000
BB(NSNB,NS)=- (Z(NS)-S)*QM(NS)-QID(NSM1)/(QL(NSM1)+QL(NSM2))-QID00378000
1(NS)/QL(NSM1) 00379000
DU 56 I=3,NSM2 00380000
56 BB(NSNB,I)=- (Z(I)-S)*QM(I)-QID(I-1)/(QL(I-1)+QL(I-2))+QID(I+1)/00381000
1(QL(I+1)+QL(I)) 00382000
DU 76 I=1,NS 00383000
FGAMMA = Y(NS4P1)+GAMMA(I)*V 00384000
COSFGA(I)=DCOS(FGAMMA) 00385000
SINFGA(I)=DSIN(FGAMMA) 00386000
FALFA=Y(NS4P1)+V*ALFA(I) 00386020
SINFAL(I)=DSIN(FALFA) 00386030
COSFAL(I)=DCOS(FALFA) 00386040
QMESIN(I)=QME(I)*SINFAL(I) 00386050
QMECOS(I)=QME(I)*COSFAL(I) 00386060
BETA(I)=V*BETA(I) 00386070
QIDBET=QID(I)*BETA(I) 00386080
QIDBSN(I)=QIDBET*SINFGA(I) 00386090
76 QIDBCS(I)=QIDBET*COSFGA(I) 00386100
DU 67 J=1,NS 00387000
CC(J)=0.0 00388000
DU 67 I=1,NS 00389000
67 CC(J)=CC(J)+B(I,J)*QIDBSN(I)+C(I,J)*QME(I)*SINFAL(I) 00406000
CC(IB1)=0.0 00407000
CC(IBNB)=0.0 00408000
DU 78 I=1,NS 00409000
QMZ(I)=Q-Z(I) 00410000
CIB1=QMZ(I)*QMESIN(I)-QIDBSN(I) 00411000
CIBNB=(Z(I)-S)*QMESIN(I)+QIDBSN(I) 00412000
CC(IB1)=CC(IB1)+CIB1 00413000

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78	CC(IBNB)=CC(IBNB)+CIBNB	00414000
	DO 71 J=NSP1,NS2	00416000
	K=J-NS	00417000
	CC(J)=0.0	00418000
	DO 71 I=1,NS	00419000
71	CC(J)=CC(J)-COSFAL(I)*C(I,K)*QME(I)-B(I,K)*QIDBCS(I)	00421000
	CC(NSB1)=0.0	00422000
	CC(NSNB)=0.0	00423000
	DO 73 I=1,NS	00424000
	CIB1=-QMZ(I)*QMECOS(I)+QIDBCS(I)	00425000
	CIBNB=-(Z(I)-S)*QMECOS(I)-QIDBCS(I)	00426000
	CC(NSB1)=CC(NSB1)+CIB1	00427000
73	CC(NSNB)=CC(NSNB)+CIBNB	00428000
	CC(NS2P1)=0.0	00430000
	DO 75 I=1,NS	00431000
	AA(NS2P1,I)=-QMESIN(I)	00432000
	BB(NS2P1,I)=QMECOS(I)	00433000
	C2NP1=QIRO(I)+QME(I)*ECC(I)	00434000
75	CC(NS2P1)=CC(NS2P1)+C2NP1	00435000
	FDOTSQ=FDOT**2	00436000
	DO 201 I=1,NS	00437000
	I3NS=I+NS3	00438000
	QMEFSQ(I)=QME(I)*FDOTSQ	00439000
	QKHDD(I)=QKHD(I)*(FDOT-XKF(I))*Y(I3NS)	00440000
	QCHDD(I)=QCHD(I)*(FDOT-XCF(I))*Y(I3NS)	00441000
	QIRFDD(I)=QIRO(I)*FDOT	00444000
	QKHDF(I)=QKHDF(I)*(FDOT-XKFF(I))*Y(I3NS)	00445000
201	QCHDF(I)=QCHDF(I)*(FDOT-XCFF(I))*Y(I3NS)	00446000
	FOSAVE=FDOT	00446500
	FDDT=FDOT-FDOFIX	00446550
	DO 202 I=1,NB	00447000
	IBI=IB(I)	00448000
	KKB=KK(IBI)	00449000
	BRO=DSQRT(XB(IBI)**2+YB(IBI)**2)	00451000
202	BRGROP(IBI)=(FDOT*(BNB(IBI,KKB)+BBB(IBI,KKB)*(BRO-BRUB(IBI,KKB)))+00452000	
	1KKB(IBI,KKB))*(BR0**(BHB(IBI,KKB)-1.))+BDB(IBI,KKB)+BEB(IBI,KKB)/	00454000
	2BR0)	00455000
	FDOT=FOSAVE	00455500
	DO 63 I=1,NS	00456000
63	A(I)=0.0	00457000
	DO 54 I=1,NB	00458000
	IBI=IB(I)	00459000

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54      A( IBI)=BRGROP( IBI)*XB( IBI)+BCB( IBI)*XBDOT( IBI)                00460000
      DO 203 I=1,NS                                                         00461000
203     AFX( I)=- (QM( I)*GX-QMEFSQ( I)*COSFAL( I)+QK( I)*XX( I)+QC( I)*XDOT( I00462000
      1)+QKP( I)*YY( I)+QCP( I)*YDOT( I)+QKHDD( I)*YY( I)+QCHDD( I)*YDOT( I) )-A( I00463000
      2) .                                                                    00464000
      DO 204 I=1,NB                                                         00465000
      IBI=IB( I)                                                            00466000
204     A( IBI)=BRGROP( IBI)*YB( IBI)+BCB( IBI)*YSDOT( IBI)              00467000
      DO 205 I=1,NS                                                         00468000
      TRIG=( QID( I)-QIRO( I))*FDOTSQ*BETA( I)                             00468010
      AFY( I)=- (QM( I)*GY-QMEFSQ( I)*SINFAL( I)+QK( I)*YY( I)+QC( I)*YDOT( I00469000
      1)-QKP( I)*XX( I)-QCP( I)*XDOT( I)-QKHDD( I)*XX( I)-QCHDD( I)*XDOT( I) )-A( I00470000
      2) .                                                                    00471000
      AMY1( I)=TRIG*CUSFGA( I)                                             00472000
205     AMX1( I)=-TRIG*SINFGA( I)                                          00473000
      I=1                                                                    00474000
      J=2                                                                    00475000
      K=1                                                                    00476000
57     AMY2( I)=-1./QL( K)*( QIRFDO( I)*( YDOT( J)-YDOT( K))+QKF( I)*( XX( J)-X00477000
      1X( K) )+                                                                00478000
      1   QCF( I)*( XDOT( J)-XDOT( K))+QKPF( I)*( YY( J)-YY( K))+QCPF( I)*( YDOT( I00479000
      2J)-YDOT( K))+QKHDF( I)*( YY( J)-YY( K))+QCHDF( I)*( YDOT( J)-YDOT( K) ) ) 00480000
      AMX2( I)=1./QL( K)*( -QIRFDO( I)*( XDOT( J)-XDOT( K))+QKF( I)*( YY( J)-Y00481000
      1Y( K) )+QCF( I)*( YDOT( J)-YDOT( K))-QKPF( I)*( XX( J)-XX( K))-QCPF( I)*( XDOT00482000
      2( J)-XDOT( K))-QKHDF( I)*( XX( J)-XX( K))-QCHDF( I)*( XDOT( J)-XDOT( K) ) ) 00483000
      IF ( I.GT.1) GO TO 58
      I=NS                                                                    00484000
      J=NS                                                                    00485000
      K=NSM1                                                                  00486000
      GO TO 57                                                                00487000
      GO TO 57                                                                00488000
58     DO 59 I=2,NSM1                                                       00490000
      AMY2( I)= -1./ (QL( I-1)+QL( I))*( QIRFDO( I)*( YDOT( I+1)-YDOT( I-1))+00491000
      1QKF( I)*( XX( I+1)-XX( I-1))+QCF( I)*( XDOT( I+1)-XDOT( I-1))+QKPF( I)*( YY( I00492000
      2I+1)-YY( I-1))+QCPF( I)*( YDOT( I+1)-YDOT( I-1))+QKHDF( I)*( YY( I+1)-YY( I00493000
      3I-1))+QCHDF( I)*( YDOT( I+1)-YDOT( I-1) ) )                          00494000
59     AMX2( I)=1./ (QL( I-1)+QL( I))*( -QIRFDO( I)*( XDOT( I+1)-XDOT( I-1))+000495000
      1KF( I)*( YY( I+1)-YY( I-1))+QCF( I)*( XDOT( I+1)-XDOT( I-1))-QKPF( I)*( XX( I00496000
      2+1)-XX( I-1))-QCPF( I)*( XDOT( I+1)-XDOT( I-1))-QKHDF( I)*( XX( I+1)-XX( I00497000
      3-1))-QCHDF( I)*( XDOT( I+1)-XDOT( I-1) ) )                          00498000
      DO 60 I=1,NS                                                         00499000
      AMY( I)=AMY1( I)+AMY2( I)                                             00500000
60     AMX( I)=AMX1( I)+AMX2( I)                                           00501000

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181

	DU 69 J=1,NS	00502000
	DST=0.	00503000
	DO 44 I=1,NS	00504000
	DSTAT = -C(I,J)*AFX(I)-B(I,J)*AMY(I)	00505000
44	DST=DST+DSTAT	00506000
69	DSTA(J)=DST-XX(IB1)-(XX(IBNB)-XX(IB1))*(Z(J)-S)/QLL+XX(J)	00507000
	DSTA(IB1)=0.0	00508000
	DSTA(IBNB)=0.0	00509000
	DO 43 I=1,NS	00510000
	DST1 = -(Q-Z(I))*AFX(I)-AMY(I)	00511000
	DSTA(IB1)=DSTA(IB1)+DST1	00512000
	DSTNB=-(Z(I)-S)*AFX(I)-AMY(I)	00513000
43	DSTA(IBNB)=DSTA(IBNB)+DSTNB	00514000
	DU 42 J=NSP1,NS2	00517000
	K=J-NS	00518000
	DST=0.	00519000
	DO 41 I=1,NS	00520000
	DSTAT=-C(I,K)*AFY(I)+B(I,K)*AMX(I)	00521000
41	DST=DST+DSTAT	00522000
42	DSTA(J)=DST-YY(IB1)-(YY(IBNB)-YY(IB1))*(Z(K)-S)/QLL+YY(K)	00523000
	DSTA(NSB1)=0.0	00524000
	DSTA(NSNB)=0.0	00525000
	DO 70 I=1,NS	00526000
	DSTN1=-(Q-Z(I))*AFY(I)-AMX(I)	00527000
	DSTA(NSB1)=DSTA(NSB1)+DSTN1	00528000
	DSTNB=-(Z(I)-S)*AFY(I)+AMX(I)	00529000
70	DSTA(NSNB)=DSTA(NSNB)+DSTNB	00530000
	DSTA(NS2P1)= TMZ1*FDOT**TMZ+TMZ2*FDOT+TMZ3	00532000
	DO 72 I=1,NS	00533000
	DSR=-(QME(I))*(GY*COSFAL(I)-GX*SINFAL(I))+(CZ1(I)*FDOT**CZ(I)+	00534000
	1CZ2(I)*FDOT)	00535000
72	DSTA(NS2P1)=DSTA(NS2P1)+DSR	00536000
	DO 172 J=1,NS	00537000
	JNS=J+NS	00538000
	DO 172 I=1,NS	00539000
	INS=I+NS	00540000
	AROT(J,I)=AA(J,I)*COSAB(I)	00541000
	AROT(J,INS)=-AA(J,I)*SINAB(I)*Y(I)	00542000
	AROT(JNS,I)= BB(JNS,I)*SINAB(I)	00543000
	AROT(JNS,INS)=BB(JNS,I)*COSAB(I)*Y(I)	00544000
	AROT(NS2P1,I)=AA(NS2P1,I)*COSAB(I)+BB(NS2P1,I)*SINAB(I)	00546000
172	AROT(NS2P1,INS)=-AA(NS2P1,I)*SINAB(I)*Y(I)+BB(NS2P1,I)*COSAB(I)*Y	00547000

	11)		00548000
		DO 173 J=1,NS	00549000
		TRIG=0.	00550000
		DO 174 I=1,NS	00551000
		INS =I+NS	00552000
		.I2NS=I+NS2	00553000
		I3NS=I+NS3	00554000
		DRDT=AA(J,I)*(YDUT(I)*Y(I3NS)+Y(I2NS)*Y(I3NS)*SINAB(I))	00555000
	174	TRIG=TRIG+DRDT	00556000
	173	DRO(J)=TRIG+DSTA(J)	00557000
		DO 175 J=NSP1,NS2	00558000
		TRIG=0.	00560000
		DO 176 I=1,NS	00561000
		INS=I+NS	00562000
		I2NS=I+NS2	00563000
		I3NS=I+NS3	00564000
		DRDT=-BB(J,I)*(XDOT(I)*Y(I3NS)+Y(I2NS)*Y(I3NS)*COSAB(I))	00565000
	176	TRIG=TRIG+DRDT	00566000
	175	DRO(J)=TRIG+DSTA(J)	00567000
		DRO(NS2P1)=DSTA(NS2P1)	00569000
		DO 177 I=1,NS	00570000
		INS=I+NS	00571000
		I2NS=I+NS2	00572000
		I3NS=I+NS3	00573000
		DRDT=AA(NS2P1,I)*(YDUT(I)*Y(I3NS)+Y(I2NS)*Y(I3NS)*SINAB(I))-BB	00574000
		1(NS2P1,I)*(XDOT(I)*Y(I3NS)+Y(I2NS)*Y(I3NS)*COSAB(I))	00575000
	177	DRO(NS2P1)=DRO(NS2P1)+DRDT	00576000
		DO 39 J=1,NS2P1	00577000
	39	AROT(J,NS2P1)=CC(J)	00578000
		FF=0.0	00579000
		KL=ISIMDD(51,NS2P1,1,AROT,DRO,FF,IA)	00581000
		DO 600 I=1,NS	00582000
		INS=I+NS	00583000
		I2NS=I+NS2	00584000
		I3NS=I+NS3	00585000
		BD(I)=Y(I2NS)	00586000
		BD(INS)=Y(I3NS)	00587000
		BD(I2NS)=AROT(I,1)	00588000
	600	BD(I3NS)=AROT(INS,1)	00589000
		BD(NS4P1)=Y(NS4P2)	00590000
		BD(NS4P2)=AROT(NS2P1,1)	00591000
		WRITE (6,620) (BD(I),I=1,N)	00592000

620	FORMAT (1H1//14X,33HTHE APPROPRIATE DERIVATIVES	BD,S// (1P7D15.6)	00593000
	1)		00594000
	IF (KL.NE.3) GO TO 560		00595000
	WRITE (6,540) KL		00596000
540	FORMAT(1H1/10X,4HKL =11)		00597000
	CALL EXIT		00598000
560	RETURN		00599000
	END		00600000

Card Count 344

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SUBROUTINE BRGXY                                00223020
IMPLICIT REAL*8 (A-H,O-Z)                       00223030
COMMON/MAFU14/ FDOFIX                           00223035
COMMON/MAFU1/DUM1(75),NS,NB,IB(12)              00223040
X          /MAFU0/DUM1(2),KK(25)                 00223045
1          /MAFU5/DUM2(300),BKM(25),BKMY(25),BCM(25),BCMY(25),BCB(25) 00223050
2          /MAFU6/BNB(25,6),BBB(25,6),BROB(25,7),BKB(25,6),RHB(25,6), 00223060
3BDB(25,6),BEB(25,6)                             00223070
4          /MAFU8/TULB                            00223080
5          /MAFU11/FD,FDT                         00223090
6          /GARBG7/XX(25),YY(25),XB(25),YB(25),XDUT(25),YDOT(25), 00223100
7XBDUT(25),YBDUT(25)                             00223110
8          /GARBG8/XBO,XB1,XB2,YBO,YB1,YB2,BROXO,BROX1,BRUX2,BROYO, 00223120
9BROY1,BROY2,FBRGX0,FBRGX1,FBRGX2,FBRGYO,FBRGY1,FBRGY2,I,IBI,KKB 00223130
          FUSAVE=FDOT
          FDOT=FDOT-FDOFIX                       00223200
10 DU 162 I=1,NB                                  00223300
          IBI=IB(I)                              00224000
          IF (BCM(X(IBI)).NE.0) GO TO 91           00225000
          BCM(X(IBI))=1.D-16                      00226000
91          XBDOT(IBI)=XDUT(IBI)/(1+BCB(IBI)/BCM(X(IBI))) 00227000
          IF (BCMY(IBI).NE.0) GO TO 101          00228000
          BCM(Y(IBI))=1.D-16                     00229000
101         YBDOT(IBI)=YDOT(IBI)/(1+BCR(IBI)/BCMY(Y(IBI))) 00230000
          IF (BCB(IBI).EQ.0.)BCB(IBI)=1.D-16    00231000
          KKB=KK(IBI)                             00234000
          XB2=0.3D0*XX(IBI)                       00240000
          XB1=0.1D0*XX(IBI)                       00241000
          BROX2=DSQRT(XB2**2+(YY(IBI)/(BKM(X(IBI))/BKMY(Y(IBI))*XX(IBI)/XB2-1.)) 00242000
          +1.)**2)                                00243000
          BROX1=DSQRT(XB1**2+(YY(IBI)/(BKM(X(IBI))/BKMY(Y(IBI))*XX(IBI)/XB1-1.)) 00244000
          +1.)**2)                                00245000
          FBRGX2=(FDOT*(BNB(IBI,KKB)+BBB(IBI,KKB)*(BROX2-BROB(IBI,KKB))) + 00246000
1BKB(IBI,KKB))*(BROX2**2*(BHB(IBI,KKB)-1.)+BDB(IBI,KKB)+BEB(IBI,KKB)/00248000
2BROX2)**XB2-BKM(X(IBI))*(XX(IBI)-XB2)          00249000
          FBRGX1=(FDOT*(BNB(IBI,KKB)+BBB(IBI,KKB)*(BROX1-BROB(IBI,KKB))) + 00250000
1BKB(IBI,KKB))*(BROX1**2*(BHB(IBI,KKB)-1.)+BDB(IBI,KKB)+BEB(IBI,KKB)/00251000
2BROX1)**XB1-BKM(X(IBI))*(XX(IBI)-XB1)          00252000
52         XBO=XB1-FBRGX1/(FBRGX2-FBRGX1)*(XB2-XB1) 00253000
          BROXO=DSQRT(XBO**2+(YY(IBI)/(BKM(X(IBI))/BKMY(Y(IBI))*XX(IBI)/XBO-1.)) 00254000
          +1.)**2)                                00255000
          FBRGXO=(FDOT*(BNB(IBI,KKB)+BBB(IBI,KKB)*(BROXO-BROB(IBI,KKB))) + 00256000

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1BKB (IBI, KKB) *(BRUXO** (BHB (IBI, KKB) -1.) +BDR (IBI, KKB) +BEB (IBI, KKB) /00257000
2BRUXO)*XBO-BKMX (IBI) *(XX (IBI) -XBO) 00258000
PERCEX=XBO/XX (IBI) 00258500
IF (DABS (1.-PERCEX) .LE. 1.D-8) GOTO152 00259000
IF (DABS (PERCEX) .LT. 1.D-8) GOTO152 00259500
PERCEX=DABS (FBRGXO/(BKMX (IBI) *(XX (IBI) -XBO))) 00260000
IF (PERCEX.LT.TOLB) GO TO 152 00265000
IF (DABS (XB2) .GE. DABS (XB1)) XB2=XB1 00265500
XB1=XBO 00267000
FBRGX2=FBRGX1 00268000
FBRGX1=FBRGXO 00269000
GO TO 52 00270000
152 XB (IBI)=XBO 00271000
YB2=0.3D0*YY (IBI) 00275000
YB1=0.1D0*YY (IBI) 00276000
BROY2=DSQRT (YB2**2+(XX (IBI) / (BKMY (IBI) /BKMX (IBI) *(YY (IBI) /YB2-1.)
1+1.) **2) 00277000
BROY1=DSQRT (YB1**2+(XX (IBI) / (BKMY (IBI) /BKMX (IBI) *(YY (IBI) /YB1-1.)
1+1.) **2) 00278000
00279000
00280000
FBRGY2=(FDOT*(BNB (IBI, KKB) +BBB (IBI, KKB) *(BROY2-BROB (IBI, KKB))) +
1BKB (IBI, KKB) *(BROY2** (BHB (IBI, KKB) -1.) +BDB (IBI, KKB) +BEB (IBI, KKB) /00282000
2BROY2)*YB2-BKMY (IBI) *(YY (IBI) -YB2) 00283000
FBRGY1=(FDOT*(BNB (IBI, KKB) +BBB (IBI, KKB) *(BROY1-BROB (IBI, KKB))) +
1BKB (IBI, KKB) *(BROY1** (BHB (IBI, KKB) -1.) +BDR (IBI, KKB) +BEB (IBI, KKB) /00285000
2BROY1)*YB1-BKMY (IBI) *(YY (IBI) -YB1) 00286000
64 YB0=YB1-FBRGY1/(FBRGY2-FBRGY1)*(YB2-YB1) 00287000
BROY0=DSQRT (YB0**2+(XX (IBI) / (BKMY (IBI) /BKMX (IBI) *(YY (IBI) /YB0-1.)
1+1.) **2) 00288000
00289000
FBRGY0=(FDOT*(BNB (IBI, KKB) +BBB (IBI, KKB) *(BROY0-BROB (IBI, KKB))) +
1BKB (IBI, KKB) *(BROY0** (BHB (IBI, KKB) -1.) +BDB (IBI, KKB) +BEB (IBI, KKB) /00291000
2BROY0)*YB0-BKMY (IBI) *(YY (IBI) -YB0) 00292000
PERCEY=YB0/YY (IBI) 00292500
IF (DABS (1.-PERCEY) .LE. 1.D-8) GOTO162 00293000
IF (DABS (PERCEY) .LT. 1.D-8) GOTO162 00293500
PERCEY=DABS (FBRGYO/(BKMY (IBI) *(YY (IBI) -YB0))) 00294000
IF (PERCEY.LT.TOLB) GO TO 162 00299000
IF (DABS (YB2) .GE. DABS (YB1)) YB2=YB1 00300000
YB1=YB0 00301000
FBRGY2=FBRGY1 00302000
FBRGY1=FBRGYO 00303000
GO TO 64 00304000
162 YB (IBI)=YB0 00305000
FDDT=FOSAVE 00305500
RETURN 00306000
END 00307000

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SUBROUTINE INFLCO (C,B)                                00601000
IMPLICIT REAL*8 (A-H,O-Z)                            00602000
REAL*8 MOF,MOM                                       00603000
REAL Z                                               00604000
DIMENSION DD(25), D(25), QL(25),EE(25), GG(25), EIO(25), GAK(25), 00605000
1SZ(25),ZQ(25),SZOL(25),ZQQL(25),QZ(25),ZSUL(25),QZOL(25),IB(12), 00606000
2SHERGA(25),ROF(25,25),ROM(25,25),AFALEI(25),QLEI(25),SQL2EI(25) 00606010
2,C(25,25),B(25,25), Z(25), FOF(25,25), MOF(25,25), FOM(25,25) 00607000
3),MOM(25,25)                                       00608000
COMMON/MAFUF1/DD,D,QL, NS,NB,IB                    00613000
COMMON/MAFUF2/ SZ,QZ,ZQ,SZOL,ZSUL,QZOL,ZQOL,QMLDV,QLL 00614000
COMMON/MAFUF3/ IB1,IB2                            00615000
COMMON/MAFUF4/ Z                                    00616000
COMMON/MAFC/ EE,GG,EIO,GAK                        00617000
1 /SFP2/NSM1,IDUM1(9)                              00617030
2 /GARBG2/FOF,MUF,FOM,MOM,QLEI,SQL2EI,ROF,ROM,AFALEI,SHERGA,A, 00617060
3E,F,G,GAQL,THETAM,THETAF,DSQ,DDSQ,D1,D2,I,J,K    00617090
DATA PI/3.1415926535897932/                       00617120
A=PI/4.                                             00619000
E=A/16.                                             00620000
F=4./3.D0                                          00621000
G=F/2.                                             00622000
DD 200 J=1,NSM1                                    00624000
DSQ=D(J)**2                                        00624500
DDSQ=DD(J)**2                                      00625000
D1=DDSQ-DSQ                                        00625500
D2=DDSQ+DSQ                                        00626000
QLEI(J)=QL(J)/(E*D1*D2*EE(J)+EIO(J))             00626500
SQL2EI(J)=0.5* QL(J)* QLEI(J)                    00627000
GAQL=GAK(J)/QL(J)+A*GG(J)*D1/(QL(J)*(F*(DDSQ*DD(J)-DSQ*D(J))/( D2*
1(DD(J)-D(J))))))                                00631000
SHERGA(J)=1./GAQL                                  00632000
200 AFALEI(J)=SHERGA(J)+G*QL(J)*SQL2EI(J)         00633000
DO 400 I=1,NS                                      00642000
DO 400 J=1,NS                                      00643000
FOF(I,J)=0.                                        00643200
MOF(I,J)=0.                                        00643400
FOM(I,J)=0.                                        00643600
MOM(I,J)=0.                                        00643800
C(I,J)=0                                           00644000
400 B(I,J)=0                                        00645000
DO 140 I=1,IB1                                     00646000

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	IF (IB1.EQ.1) GO TO 122	00647000
112	K=I+1	00654000
	DO 120 J=K, IB1	00655000
	FOF(I,J) = -1.0	00656000
	MOF(I,J) = Z(J) - Z(I)	00657000
120	MOM(I,J) = -1.0	00659000
122	K=IB1+1	00660000
	DO 130 J=K, IB2	00661000
	FOF(I,J) = SZOL(I)	00662000
	MOF(I,J) = QZ(J) * SZOL(I)	00663000
	FOM(I,J) = QMLDV	00664000
130	MOM(I,J) = ZQUL(J)	00665000
140	CONTINUE	00673000
	KK=IB1+1	00674000
	DO 440 I=KK, IB2	00675000
412	K=IB1+1	00682000
	DO 420 J=K, I	00683000
	FOF(I,J) = QZOL(I)	00684000
	MOF(I,J) = SZ(J) * QZOL(I)	00685000
	FOM(I,J) = QMLDV	00686000
420	MOM(I,J) = ZSOL(J)	00687000
881	K=I+1	00688000
	IF (K.GT.IB2) GOTO 440	00689000
	DO 430 J=K, IB2	00690000
	FOF(I,J) = SZOL(I)	00691000
	MOF(I,J) = -ZQ(J) * SZOL(I)	00692000
	FOM(I,J) = QMLDV	00693000
430	MOM(I,J) = ZQUL(J)	00694000
440	CONTINUE	00702000
	IF (IB2.EQ.NS) GO TO 542	00703000
	KK=IB2+1	00704000
	DO 540 I=KK, NS	00705000
512	K=IB1+1	00712000
	DO 520 J=K, IB2	00713000
	FOF(I,J) = QZOL(I)	00714000
	MOF(I,J) = SZ(J) * QZOL(I)	00715000
	FOM(I,J) = QMLDV	00716000
520	MOM(I,J) = ZSOL(J)	00717000
	K=IB2+1	00718000
	DO 530 J=K, I	00719000
	FOF(I,J) = 1.	00720000
	MOF(I,J) = Z(I) - Z(J)	00721000

530	MOM(I,J) =1.0	00723000
540	CONTINUE	00731000
542	DO 600 I=1,NS	00732000
	ROF(I,1)=0.	00732200
	ROM(I,1)=0.	00732400
	THETA F=0.	00732600
	THETA M=0.	00732800
	DO 600 J=2,NS	00733000
	ROF(I,J)=ROF(I,J-1)+QL(J-1)*THETA F+AFAL EI(J-1)*FOF(I,J)+MOF(I,J)*	00734000
	1SQL2EI(J-1)	00735000
	ROM(I,J)=ROM(I,J-1)+QL(J-1)*THETA M+AFAL EI(J-1)*FOM(I,J)+MOM(I,J)*	00736000
	1SQL2EI(J-1)	00737000
	THETA F=THETA F+SQL2EI(J-1)*FOF(I,J)+QLEI(J-1)*MOF(I,J)	00738000
600	THETA M=THETA M+SQL2EI(J-1)*FOM(I,J)+QLEI(J-1)*MOM(I,J)	00739000
	DO 700 I=1,NS	00742000
	DO 700 J=1,NS	00743000
	C(I,J)=ROF(I,J)-ROF(I,IB1)+SZOL(J)*(ROF(I,IB2)-ROF(I,IB1))	00748000
700	B(I,J)=ROM(I,J)-ROM(I,IB1)+SZOL(J)*(ROM(I,IB2)-ROM(I,IB1))	00749000
	RETURN	00750000
	END	00751000

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Card Count 104

	SUBROUTINE SZMASS	01496000
	IMPLICIT REAL*8 (A-H,O-Z)	01497000
	REAL Z	01498000
	DIMENSION Z(25),QL(25),SZ(25),QZ(25),ZQ(25),SZOL(25),ZSOL(25),	001500000
	1ZOL(25),ZQOL(25), DD(25),D(25),DN(25), Q6LDND(25),DDPLD(25),DDL(201501000	
	25),Q1LDND(25), QM(25),AM(25),QID(25),AID(25),QIRU(25),AIRO(25)	01502000
	DIMENSION ECC(25),ALFA(25),BETA(25),GAMMA(25)	01503000
	1,IB(12)	01503100
	COMMON/MAFU2/ DN,AM,AID,AIRO,ECC,ALFA,BETA,GAMMA,GX,GY	01504000
	COMMON/MAFU12/ QM,QID,QIRO	01505000
	1 /SFP2/NSM1, IDUM1(9)	01505100
	2 /GARBG1/QMASS,QLDNDD,QL2,Q1LDND,DDL,DDPLD,Q6LDND,D2,	01505200
	3DD2,WEIT,POLARA,I	01505300
	COMMON/MAFUF1/DD,D,QL,NS,NB,IB	01506000
	COMMON/MAFUF2/SZ,QZ,ZQ,SZOL,ZSOL,QZOL,ZQOL,QMLOV,QLL	01507000
	COMMON/MAFUF3/ IB1,IBNB	01508000
	COMMON/MAFUF4/Z/MAIFU/Q,S	01509000
	DATA PI/3.1415926535897932/	01510000
	U=4./3.DO	01512000
	W=PI/(128.*386.088)	01514000
	E=16.*W	01515000
	Z(1)=0	01518000
	DO 105 I=2,NS	01519000
105	Z(I)=Z(I-1)+QL(I-1)	01520000
	Q=Z(IBNB)	01522000
	S=Z(IB1)	01521000
	DO 103 I=1,NS	01523000
	SZ(I)=Z(IB1)-Z(I)	01525000
	ZQ(I)=Z(I)-Z(IBNB)	01526000
103	QZ(I)=Z-QZ(I)	01527000
	QLL=Z(IBNB)-Z(IB1)	01528000
	QMLOV=-1./QLL	01530000
	DO 104 I=1,NS	01531000
	SZOL(I)=SZ(I)/QLL	01532000
	ZSOL(I)=-SZOL(I)	01533000
	QZOL(I)=QZ(I)/QLL	01534000
104	ZQOL(I)=-QZOL(I)	01535000
	DO 50 I=1,NS	01536000
	DD2 = DD(I)**2	01537000
	D2 = D(I)**2	01538000
	QL2 = QL(I)**2	01541000
	QLDNDD =QL(I)* DN(I)*(DD2 -D2)	01542000

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Q6LDND(I) =                W*QLDNDD                01543000
DDPLD(I) = DD2      +D2                01544000
DDL(I) = DDPLD(I)+                U*QL2                01545000
50 Q1LDND(I) =                E*QLDNDD                01546000
   QM(1)=Q1LDND(1) + AM(1)                01547000
   QM(NS)=Q1LDND(NS-1) +AM(NS)            01548000
   QID(1) = Q6LDND(1)*DDL(1) + AID(1)      01549000
   QID(NS)= Q6LDND(NS-1)*DDL(NS-1)  +AID(NS) 01550000
   QIRO(1)=2.*Q6LDND(1)*DDPLD(1)+AIRO(1) 01551000
   QIRO(NS)=2.*Q6LDND(NS-1)*DDPLD(NS-1) +AIRO(NS) 01552000
   DO 55 I=2,NSM1                        01554000
   QM(I) =Q1LDND(I-1)+Q1LDND(I) +      AM(I) 01555000
   QID(I)=Q6LDND(I-1)*DDL(I-1)+Q6LDND(I)*DDL(I)+AID(I) 01556000
55 QIRO(I) = 2.*(Q6LDND(I-1)*DDPLD(I-1)+Q6LDND(I)*DDPLD(I)) +AIRO(I) 01557000
62 QMASS=0.0                              01558000
   POLARA=0.0                              01559000
   DO 40 I=1,NS                            01560000
   QMASS=QMASS+QM(I)                       01561000
40 POLARA=POLARA+QIRO(I)                   01562000
   WEIT =386.088*QMASS                     01563000
   WRITE (6,77) WEIT,QMASS,POLARA           01564000
77 FORMAT(1H0///,9X'TOTAL ROTOR WEIGHT, LB ='1PD13.4/ 9X,33HTOTAL ROTOR01565000
1R MASS,(LB*SEC**2)/IN =1PD13.4/ 9X,57HTOTAL ROTOR POLAR MASS MOMEN01566000
2T OF INERTIA, LB*IN*SEC**2 =1PD13.4/)     01567000
   RETURN                                   01568000
   END                                       01569000

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Card Count 68

	SUBROUTINE PLUT1(F)	00752000
	REAL F(50,12), Y(2)	00753000
	COMMON/SFP1/RPM(50),NPOINT	00754000
	1 /GARBG4/Y,I,J	00755100
	2 /MAFUF1/DUM1(151),NB,IB(12)	00755200
C	GET LIMITS OF F & RPM	00757000
	Y(1)=1.E70	00760000
	Y(2)=-1.E70	00761000
	DO 10 I=1,NB	00762000
	DO 10 J=1,NPOINT	00763000
	Y(1)=AMIN1(Y(1),F(J,I))	00764000
10	Y(2)=AMAX1(Y(2),F(J,I))	00765000
	CALL LRANGE(RPM(1),RPM(NPOINT),Y(1),Y(2))	00767000
C	PLOT EACH OF THE NB FUNCTIONS	00768000
	DO 20 I=1,NB	00769000
	CALL LRCURV(RPM,F(1,I),NPOINT,2,Y,0.)	00769100
	CALL LRCNVT(IB(I),1,Y,1,3,0)	00769200
	DO 20 J=1,NPOINT	00769300
20	CALL LRLABL(Y,3,0,RPM(J),F(J,I),0.)	00769400
	RETURN	00770100
	END	00771000

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Card Count 21

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C SOLUTION TO A SYSTEM OF 1ST ORDER ORDINARY DIFFERENTIAL EQUATIONS RKAD0010
C OF THE INITIAL VALUE TYPE. THE FOLLOWING METHODS ARE AVAILABLE---RKAD0020
C 1. ADAMS-MOULTON PREDICTOR-CORRECTOR FIXED INCREMENT RKAD0030
C 2. ADAMS-MOULTON PREDICTOR-CORRECTOR VARIABLE INCREMENT RKAD0040
C 3. RUNGE-KUTTA (ALSO USED TO GENERATE STARTING VALUES FOR A-M METHDS)RKAD0050
SUBROUTINE RKADAM(N,T,Y,H,IND,ITIM,TOL,NERR)
IMPLICIT REAL*8 (A-H,O-Z) RKAD0070
REAL*8 T,Y,H,TOL RKAD0080
DIMENSION Y(1),F(102,7),YB(102,5),A(102,4),YSAVE(102) RKAD0090
C N ORDER OF SYSTEM (IF REDIMENSIONING REQUIRED, CHANGE NN IN DATA RKAD0100
C STATEMENT AND ALSO THE 1ST SUBSCRIPT OF F AND YB IN DIMENSION RKAD0110
C STATEMENT). FROM CALLING PROGRAMRKAD0115
C T INDEPENDENT VARIABLE -- UPON ENTRY TO RKADAM FROM CALLING RKAD0120
C PROGRAM X IS AT BEGINNING OF STEP. UPON RETURN TO CALLING RKAD0130
C PROGRAM X IS AT END OF STEP. RKAD0140
C Y SOLUTION VECTOR OF DEPENDENT VARIABLES AS A FUNCTION OF X RKAD0150
C H INCREMENT(ALGEBRAIC) -- UPON ENTRY TO RKADAM FROM CALLING RKAD0160
C PROGRAM H IS THE TRIAL INCREMENT FOR THIS STEP. UPON RETURN TORKAD0170
C CALLING PROGRAM H IS THE TRIAL INCREMENT FOR THE NEXT STEP. RKAD0180
C IND FLAG TO SELECT METHOD FROM CALLING PROGRAMRKAD0190
C =0 ADAMS-MOULTON PREDICTOR-CORRECTOR VARIABLE INCREMENT H RKAD0200
C =1 RUNGE-KUTTA FIXED INCREMENT H RKAD0210
C =2 ADAMS-MOULTON FIXED INCREMENT H RKAD0220
C ITIM RESTART FLAG (APPLIES ONLY FOR IND=0,2) FROM CALLING PROGRAMRKAD0230
C =+1 RESTART WITH FORWARDS(IN THE DIRECTION OF SIGN(H)) RKAD0240
C INTEGRATION BY RUNGE-KUTTA TO GET STARTING VALUES RKAD0250
C =-1 SAME AS +1 EXCEPT USES BACKWARDS (-SIGN(H)) INTEGRATION RKAD0260
C =0 CONTINUE INTEGRATING RKAD0270
C TOL APPLIES ONLY TO IND=0. ALLOWABLE RELATIVE ERROR BETWEEN THE RKAD0280
C PREDICTED AND CORRECTED SOLUTIONS. FROM CALLING PRUGAMRKAD0290
C NERR ERROR FLAG RETURNED TO CALLING PROGRAM RKAD0300
C =0 SOLUTION IS VALID RKAD0310
C =1 SOLUTION INVALID -- N IS INVALID OR ELSE H HAS GONE TO 0 RKAD0320
DATA NN/102/ RKAD0330
NERR=0 RKAD0340
DO 200 I=1,N RKAD0341
200 YSAVE(I)=Y(I) RKAD0342
IF(IND.EQ.1)GOTO10 RKAD0350
IF(ITIM.EQ.0)GOTO170 RKAD0360
10 NS=0 RKAD0370
CALL OVERFL(K) RKAD0375
C TEST VALIDITY OF N RKAD0380

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	IF(N.GE.1.AND.N.LE.NN)GOTO40	RKAD0390
	20 NERR=1	RKAD0400
	30 DO 220 I=1,N	RKAD0401
	220 Y(I)=0.5*(Y(I)+YSAVE(I))	RKAD0402
	RETURN	RKAD0411
	40 IF(IND.NE.1)GOTO50	RKAD0420
C	RUNGE-KUTTA	RKAD0430
	CALL RUNKUT(N,H,T,Y,Y,A)	RKAD0440
	T=T+H	RKAD0450
	GOTO30.	RKAD0460
C	ADAMS-MOULTON	RKAD0470
	50 IF(ITIM.LT.0)GOTO180	RKAD0480
C	RESTART FORWARDS INTEGRATION (IN DIRECTION OF X+H	RKAD0490
	ISTFLG=1	RKAD0500
	IF(IND.EQ.2)ISTFLG=0	RKAD0505
	XB=T	RKAD0510
	DO60I=1,N	RKAD0520
	60 YB(I,1)=Y(I)	RKAD0530
	L=0	RKAD0535
	70 DO80I=1,3	RKAD0540
	T=XB+H*(I-1)	RKAD0560
	J=5-I	RKAD0567
	CALL RUNKUT(N,H,T,Y,YB(1,I+1),A)	RKAD0570
	DO 75 K=1,N	RKAD0575
	IF(DABS(YB(K,I+1)).LE.1.D12)GOTO73	RKAD0577
	H=H/10.	RKAD0579
	GOTO 83	RKAD0581
	73 IF(L.NE.0.AND.I.EQ.1)GOTO75	RKAD0583
	F(K,J)=A(K,1)	RKAD0585
	75 Y(K)=YB(K,I+1)	RKAD0590
	80 CONTINUE	RKAD0595
	T=T+H	RKAD0605
	CALL FUNN(N,T,Y,F)	RKAD0607,
	DO 2 I=1,N	RKAD0608
	IF(Y(I).GE.1.D14) GO TO 3	RKAD0609
	2 CONTINUE	RKAD0610
	GO TO 4	RKAD0611
	3 RETURN	RKAD0612
	4 CALL ADAMLT(N,T,Y,H,IND,ITIM,TOL,NERR,ISTFLG,F,A)	RKAD0615
	IF(ISTFLG.EQ.0)GOTO110	RKAD0620
	H=H/2	RKAD0640
	83 T=XB	RKAD0645

CALL OVERFL(K)	RKAD0650
IF(K.EQ.3)GOTO100	RKAD0660
DD90I=1,N	RKAD0670
90 Y(I)=YB(I,1)	RKAD0680
L=1	RKAD0685
GOTO70	RKAD0690
100 H=0.	RKAD0700
GOTO20	RKAD0710
C FOR ITIM = +1, FEED STARTING VALUES BACK TO CALLING PROGRAM ONE ATRKAD0720	
C A TIME (RUNGE-KUTTA SOLNS 1ST 4 PTS AND THEN A-M SOLN FOR 5TH PT)	RKAD0730
110 DD120I=1,N	RKAD0740
120 YB(I,5)=Y(I)	RKAD0750
NS=2	RKAD0760
130 T=XB+H*(NS-1)	RKAD0770
DD140I=1,N	RKAD0780
140 Y(I)=YB(I,NS)	RKAD0790
NS=NS+1	RKAD0800
IF(NS.GT.5)NS=0	RKAD0810
GOTO30	RKAD0820
C CONTINUE INTEGRATION PROCEDURE (A-M)	RKAD0830
170 IF(NS)190,190,130	RKAD0840
180 ISTFLG=0	RKAD0850
190 CALL ADAMLT(N,T,Y,H,IND,ITIM,TOL,NERR,ISTFLG,F,A)	RKAD0860
GOTO30	RKAD0870
END	RKAD0880

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Card Count 109

C	4TH ORDER RUNGE-KUTTA INTEGRATION FOR A SYSTEM OF 1ST ORDER,	RUNK0010
C	ORDINARY DIFFERENTIAL EQNS.	RUNK0015
	SUBROUTINE RUNKUT(N,H,X,Y,YY,A)	RUNK0020
C	SEE DIMENSION STATEMENT FOR LIMITATION ON ORDER OF SYSTEM	RUNK0022
C	CHANGE DIMENSION AS IS REQUIRED, I.E.,A(MAXORDER,4), V(MAXORDER)	RUNK0024
	IMPLICIT REAL*8 (A-H,O-Z)	RUNK0025
	REAL*8 X,H,Y(1),YY(1)	RUNK0030
	DIMENSION A(102,4),V(102)	RUNK0040
C	N = ORDER OF SYSTEM	RUNK0050
C	H = INTEGRATION STEP	RUNK0060
C	X = INDEPENDENT VARIABLE AT BEGINNING OF STEP	RUNK0070
C	Y = VECTOR OF DEPENDENT VARIABLES AT BEGINNING OF STEP	RUNK0080
C	YY = SOLUTION VECTOR OF DEPENDENT VARIABLES AT END OF STEP	RUNK0090
	X1=H/2.	RUNK0100
	X2=X+X1	RUNK0110
	X3=X+H	RUNK0120
	CALL FUND(N,X,Y,A(1,1))	RUNK0130
	DO 2 I=1,N	RUNK0131
	IF(Y(I).GE.1.D14) GO TO 3	RUNK0132
2	CONTINUE	RUNK0133
	GO TO 4	RUNK0134
3	RETURN	RUNK0135
4	DO 10 I=1,N	RUNK0140
10	V(I)=Y(I)+X1*A(I,1)	RUNK0150
	CALL FUND(N,X2,V,A(1,2))	RUNK0160
	DO 12 I=1,N	RUNK0161
	IF(Y(I).GE.1.D14) GO TO 13	RUNK0162
12	CONTINUE	RUNK0163
	GO TO 14	RUNK0164
13	RETURN	RUNK0165
14	DO 20 I=1,N	RUNK0170
20	V(I)=Y(I)+X1*A(I,2)	RUNK0180
	CALL FUND(N,X2,V,A(1,3))	RUNK0190
	DO 22 I=1,N	RUNK0191
	IF(Y(I).GE.1.D14) GO TO 23	RUNK0192
22	CONTINUE	RUNK0193
	GO TO 24	RUNK0194
23	RETURN	RUNK0195
24	DO 30 I=1,N	RUNK0200
30	V(I)=Y(I)+H*A(I,3)	RUNK0300
	CALL FUND(N,X3,V,A(1,4))	RUNK0310
	DO 32 I=1,N	RUNK0311

```
      IF(Y(I).GE.1.D14) GO TO 33
32 CONTINUE
      GO TO 34
33 RETURN
34 DO 40 I=1,N
40 YY(I)=Y(I)+H*(A(I,1)+2.*(A(I,2)+A(I,3))+A(I,4))/6.0
      RETURN
      END
```

```
RUNK0312
RUNK0313
RUNK0314
RUNK0315
RUNK0320
RUNK0330
RUNK0340
RUNK0350
```

Card Count 50

C	RESIAR	ADAM0440
60	IC=0	ADAM0460
	IF(ITIM.GT.0)GOTO90	ADAM0470
	DO 70 I=1,N	ADAM0475
70	YP(I)=Y(I)	ADAM0477
C	GET NEW SET OF DERIVATIVES BY BACKWARDS INTEGRATIUNS (RUNGE-KUTTA)	ADAM0480
	XB=X	ADAM0490
	DD80I=1,3	ADAM0500
	CALL RUNKUT(N,-H,XB,YP,YP,A)	ADAM0510
	XB=X-H*I	ADAM0520
	DO 78 K=1,N	ADAM0530
78	F(K,I)=A(K,1)	ADAM0532
80	CONTINUE	ADAM0535
	CALL FUND(N,XB,YP,F(1,4))	ADAM0537
90	HH=H/24.DO	ADAM0540
C	PREDICTOR SOLUTION	ADAM0550
	DD100I=1,N	ADAM0560
100	YP(I)=Y(I)+HH*(55.DO*F(I,1)-59.DO*F(I,2)+37.DO*F(I,3)-9.DO*F(I,4))	ADAM0570
130	CALL FUND(N,X+H,YP,F(1,7))	ADAM0650
C	CORRECTOR SOLUTION	ADAM0660
	DD140I=1,N	ADAM0670
140	YC(I)=Y(I)+HH*(19.DO*F(I,1)-5.DO*F(I,2)+F(I,3)+9.DO*F(I,7))	ADAM0680
C	TEST FOR FIXED INCREMENT OPTION	ADAM0682
	IF(IND.EQ.0)GOTO145	ADAM0684
	X=X+H	ADAM0686
	GOTO170	ADAM0688
C	TEST RELATIVE ERROR	ADAM0690
145	S=1	ADAM0700
	DD150I=1,N	ADAM0710
	T=DABS(YC(I)-YP(I))	ADAM0720
	U=DMAX1(DABS(YC(I)),1.DO)	ADAM0730
	V=U*TOL	ADAM0740
	W=V*.02DO	ADAM0750
	IF(T.GT.V)GOTO200	ADAM0760
	IF(W.LE.T)S=0.DO	ADAM0770
150	CONTINUE	ADAM0780
	ISTFLG=0	ADAM0790
	X=X+H	ADAM0800
C	TEST IF 4 STEPS HAVE ELAPSED USING SAME INCREMENT FOR DOUBLNG TEST	ADAM0810
	IF(IC.LT.3)GOTO160	ADAM0820
	IF(S.EQ.0)GOTO170	ADAM0830
C	SET H TO 2*H FOR NEXT STEP	ADAM0840


```
      IC=-1
      H=2.DO*H
      GOTO170
160  IC=1+IC
170  DO180 I=1,N
180  Y(I)=YC(I)
      GOTO120
200  IF(ISTFLG.NE.0)GOTO120
C    HALVE THE INCREMENT AND RECOMPUTE
      H=H/2.DO
      CALL OVERFL(K)
      IF(K.NE.3)GOTO60
      H=0
      NERR=1
120  RETURN
      END
```

```
ADAM0850
ADAM0860
ADAM0870
ADAM0880
ADAM0890
ADAM0900
ADAM0910
ADAM0920
ADAM0930
ADAM0940
ADAM0950
ADAM0960
ADAM0970
ADAM0980
ADAM0990
ADAM1000
```

Card Count 100

SUBROUTINE TIME(T,H,TA,ITIM)	00000100
IMPLICIT REAL*8 (A-H,U-Z)	00000200
IF(ITIM.EQ.0) GO TO 10	00000300
TA=0.5*H	00000400
ICOUNT=1	00000500
RETURN	00000600
10 · ICOUNT=ICOUNT+1	00000700
IF(ICOUNT.EQ.2) GO TO 2	00000800
IF(ICOUNT.EQ.3) GO TO 3	00000900
IF(ICOUNT.EQ.4) GO TO 4	00001000
IF(ICOUNT.GE.5) GO TO 5	00001100
2 TA=1.25*H	00001200
RETURN	00001300
3 TA=2.125*H	00001400
RETURN	00001500
4 TA=3.0625*H	00001600
TSAVE=T	00001700
RETURN	00001800
5 TA=TA+0.5*(T-TSAVE)	00001900
TSAVE=T	00002000
ICOUNT=ICOUNT-1	00002100
RETURN	00002200
END	00002300

C	SUBPROGRAM TO SOLVE SIMULTANEOUS LINEAR EQUATIONS	AWCU0030
C	ARGUMENTS-	AWCU0040
C		AWCU0050
C	DATE- 1/13/67 MODIFIED FOR COMPILATION IN RELEASE 14	AWCU0060
C		AWCU0070
C	DSM DIMENSIONED SIZE OF COEFFICIENT MATRIX	AWCU0080
C	NE ACTUAL NUMBER OF EQUATIONS FOR THIS CALL	AWCU0090
C	NC NUMBER OF COLUMNS IN CONSTANT MATRIX	AWCU0100
C	A COEFFICIENT MATRIX	AWCU0110
C	B CONSTANT MATRIX	AWCU0120
C	DET INPUT - SCALE FACTOR, OUTPUT - FACTOR TIMES	AWCU0130
C	DETERMINANT VALUE OF COEFFICIENT MATRIX	AWCU0140
C	C TEMPORARY STORAGE FOR SUBROUTINE	AWCU0150
C	ISIMEQ RETURNS 1 IF DK, 2 IF OVFL0, 3 IF SINGULAR	AWCU0160
C	IF NC IS NEGATIVE, THE INVERSE OF THE COEFFICIENT	AWCU0170
C	MATRIX IS REQUIRED, MATRIX B IS SET UP AS IDENTITY.	AWCU0180
C	FUNCTION ISIMDD(DSM, NE, NC, A, B, DET, C)	AWCU0190
	LOGICAL DVO	AWCU0200
	INTEGER DSM, C, T, SUB1, SUB2, R, D	AWCU0210
	DOUBLE PRECISION B, PIVOT, DET, S	AWCU0220
	DIMENSION B(1),C(1)	AWCU0230
C	INITIALIZE	AWCU0240
	N = NE	AWCU0250
	D = DSM	AWCU0260
	M = IABS(NC)	AWCU0270
	ISIMDD = 1	AWCU0280
	DVO = .FALSE.	AWCU0290
	DO 1 I, = 1,N	AWCU0300
	1 C(I) = I	AWCU0310
	IF(NC) 5, 15, 15	AWCU0320
C	INVERSE REQUIRED	AWCU0330
	5 SUB2 = 0	AWCU0340
	DO 10 J = 1,N	AWCU0350
	SUB1 = SUB2	AWCU0360
	DO 6 I = 1,N	AWCU0370
	SUB1 = SUB1 + 1	AWCU0380
	6 B(SUB1) = 0.0	AWCU0390
	SUB1 = SUB2 + J	AWCU0400
	B(SUB1) = 1.000	AWCU0410
	10 SUB2 = SUB2 + D	AWCU0420
	GO TO 15	AWCU0430
	ENTRY IDETDD(DSM, NE, A, DET)	AWCU0440

	DOUBLE PRECISION A	AWCU0450
	DIMENSION A(1)	AWCU0460
	N = NE	AWCU0470
	D = DSM	AWCU0480
	IDETDD = 1	AWCU0490
	DVO = .TRUE.	AWCU0500
C	START MAIN LOOP	AWCU0510
15	DO 1000 L = 1,N	AWCU0520
	LP1 = L + 1	AWCU0530
	DO 40 I = L,N	AWCU0540
	PIVOT = 0.000	AWCU0550
	SUB1 = (L-1) * D + I	AWCU0560
	SUB2 = SUB1	AWCU0570
	DO 20 J = L,N	AWCU0580
	IF(DABS(PIVOT) .GE. DABS(A(SUB1))) GO TO 20	AWCU0590
	PIVOT = A(SUB1)	AWCU0600
	JB = J	AWCU0610
20	SUB1 = SUB1 + D	AWCU0620
C	COMPUTE DETERMINANT	AWCU0630
	CALL OVERFL(T)	AWCU0640
	DET = DET * PIVOT	AWCU0650
	IF(.NOT. DVO) GO TO 24	AWCU0660
	CALL OVERFL(T)	AWCU0670
	IF(T .EQ. 1) IDETDD = 2	AWCU0680
C	TEST FOR SINGULAR MATRIX	AWCU0690
24	IF(PIVOT .EQ. 0.000)GO TO 2000	AWCU0700
	DO 25 J = L,N	AWCU0710
	A(SUB2) = A(SUB2) / PIVOT	AWCU0720
25	SUB2 = SUB2 + D	AWCU0730
	IF (DVO) GO TO 35	AWCU0740
	SUB1 = I	AWCU0750
	DO 30 J = 1,M	AWCU0760
	B(SUB1) = B(SUB1) / PIVOT	AWCU0770
30	SUB1 = SUB1 + D	AWCU0780
35	IF (I .EQ. L) JP = JB	AWCU0790
40	CONTINUE	AWCU0800
C	INTERCHANGE COLUMNS	AWCU0810
100	IF (JP .EQ. L) GO TO 260	AWCU0820
	IF (DVO) GO TO 110	AWCU0830
	T = C(L)	AWCU0840
	C(L) = C(JP)	AWCU0850
	C(JP) = T	AWCU0860

110	R = D * L - D	AWCU0870
	T = D * JP - D	AWCU0880
	DO 120 I = 1,N	AWCU0890
	SUB1 = R + I	AWCU0900
	SUB2 = T + I	AWCU0910
	S = A(SUB1)	AWCU0920
	A(SUB1) = A(SUB2)	AWCU0930
120	A(SUB2) = S	AWCU0940
	DET = -DET	AWCU0950
C	REDUCE PIVOT COLUMN	AWCU0960
260	R = D* L - D	AWCU0970
	DO 400 I = 1,N	AWCU0980
	IP = R + I	AWCU0990
	PIVOT = A(IP)	AWCU1000
	IF (I .EQ. L .OR. PIVOT .EQ. 0.0) GO TO 400	AWCU1010
	SUB1 = L	AWCU1020
	SUB2 = I	AWCU1030
	DO 360 J = 1,N	AWCU1040
	IF (J .LT. LP1) GO TO 300	AWCU1050
	S = PIVOT * A(SUB1)	AWCU1060
	A(SUB2) = A(SUB2) - S	AWCU1070
	IF(DABS(A(SUB2)) .LT. DABS(3.0E-8*S)) A(SUB2) = 0.0	AWCU1080
300	IF (DVO .OR. J .GT. M) GO TO 350	AWCU1090
	B(SUB2) = B(SUB2) - PIVOT * B(SUB1)	AWCU1100
350	SUB1 = SUB1 + D	AWCU1110
360	SUB2 = SUB2 + D.	AWCU1120
400	CONTINUE	AWCU1130
1000	CONTINUE	AWCU1140
	IF (DVO) GO TO 1500	AWCU1150
C	REARRANGE VARIABLES	AWCU1160
1100	DO 1201 L=1,N	AWCU1170
	SUB1 = C(L)	AWCU1180
	SUB2 = L	AWCU1190
	DO 1200 J = 1,M	AWCU1200
	A(SUB1) = B(SUB2)	AWCU1210
	SUB1 = SUB1 + D	AWCU1220
1200	SUB2 = SUB2 + D	AWCU1230
1201	CONTINUE	AWCU1240
1500	RETURN	AWCU1250
C	SINGULAR COEFFICIENT MATRIX	AWCU1260
2000	IF(DVO) GO TO 3000	AWCU1270
	ISIMDD = 3	AWCU1280
	GO TO 1500	AWCU1290
3000	IDETDD = 3	AWCU1300
	GO TO 1500	AWCU1310
	END	AWCU1320

	TIME TU	TIME IN REG 0	AWCG0450
	ST 0,TZERO		AWCG0460
	L 13,SAVE+4		AWCG0470
	LM 14,12,12(13)		AWCG0480
	MVI 12(13),X'FF'		AWCG0490
	LA 15,0		AWCG0500
	BR 14	RETURN	AWCG0510
	EJECT		AWCG0520
* READ	ELAPSED TIME IN SECONDS		AWCG0530
TIMERV	EQU *		AWCG0540
TIMEV	SAVE (14,12),,TIMEV		AWCG0550
	BALR - 10,0		AWCG0560
	USING *,10		AWCG0570
	ST 13,SAVE+4		AWCG0580
	LA 2,SAVE		AWCG0590
	ST 2,8(13)		AWCG0600
	L 4,0(1)	LOC OF USERS TIME CELL	AWCG0610
	TIME TU	TIME IN REG 0	AWCG0620
	TM FSTSW,1	DID COUNTV GET CALLED	AWCG0630
206	BNZ T1		(N1) AWCG0640
	ST 0,TZERO	COUNTV WASN'T CALLED	(N1) AWCG0650
	MVI FSTSW,1 SET FLAG TO	PRETEND COUNTV NOW BEING CALLED	(N1) AWCG0660
	SER 2,2	ZERO FL PT REG	(N1) AWCG0670
	B T3		(N1) AWCG0680
	* *		(N1) AWCG0690
* T1	SL 0,TZERO	COUNTV WAS CALLED	(N1) AWCG0700
	BC 3,T2	IS DIFFERENCE NEGATIVE	(N1) AWCG0710
	AL 0,DAY	YES ADD 24 HRS	(N1) AWCG0720
T2	ST 0,BITS2	ELAPSED TIME IN REG 0	(N1) AWCG0730
	LD 2,BITS1	ELAPSED TIME IN REG 2	(N1) AWCG0740
	DD 2,FACTOR	CONVERT TIMER UNITS TO SECONDS.	(N1) AWCG0750
T3	STE 2,0(4)	STORE IN USERS TIME CELL	AWCG0760
	L 13,SAVE+4		AWCG0770
	LM 14,12,12(13)		AWCG0780
	MVI 12(13),X'FF'		AWCG0790
	LA 15,0		AWCG0800
	BR 14	RETURN	AWCG0810
	EJECT		AWCG0820
FSTSW	DC X'0'	SET FIRST TIME THRU	AWCG0830
FACTOR	DC D'38400'	300*2**7= 38400	(N1) AWCG0840
BITS1	DC X'4E000000'	1-ST HALF OF DOUBLE-WORD 'BITS1'	(N1) AWCG0850
BITS2	DC X'00000000'	2-ND HALF OF DOUBLE-WORD 'BITS1'	(N1) AWCG0860

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DAY          DC X'C5C10000' =3317760000= 38400*86400= 300*2**7*60*60*24(N1)  AWCG0870
TZERO       DC   F'0'          FIRST TIME READING .                          AWCG0880
SAVE        DC   18F'0'        SAVE AREA                                       AWCG0890
*THE TIME TU MACRO RETURNS THE TIME OF DAY AS 32-BIT UNSIGNED INTEGER.  AWCG0900
*BIT 31= 1/(300*2**7) SEC.= 26 MICRO-SEC.                                  AWCG0910
*BIT 24= 1/300 SEC.                                                       AWCG0920
*BIT 0= 2**24/300 SEC.= 15.5 HOURS                                         AWCG0930
*MAX. TIME = 2**25/300 SEC. = 31 HOURS WHICH IS ENOUGH TO CONTAIN THE  AWCG0940
*   TIME OF DAY BASED ON A 24-HOUR CLOCK.                                  AWCG0950
*MAX. RESOLUTION = 1/60 SEC. STANDARD OR 26 MICRO-SEC. FOR AN OPTIONAL  AWCG0960
*   HIGH-RESOLUTION TIMER WHICH IS NOT STANDARD AT MOST INSTALLATIONS.  AWCG0970
                                                AWCG0980
                                                AWCG0990
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