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ABSTRACT

Heat exchangers with one shell pass and n tube passes are often referred to as 1-n exchangers. The heat transfer literature contains many references to studies of 1-n exchangers when n is even but apparently little work has been done with respect to the 1-n exchanger when n is odd. This thesis greatly expands the theoretical study of 1-n exchangers with n being odd. While a completely closed form solution was found to be unfeasible, a polynomial approximation has been developed that yields the effectiveness (ε) of the two possible arrangements of the 1-3 exchanger as a function of the capacity rate ratio (R) and the number of transfer units (N_{tu}). It is also shown that the effectiveness of the arrangement with two counterflow and one parallel flow tube side passes exceeds that of some of the 1-n exchangers with n even.

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NOMENCLATURE

English Letter Symbols

A	=	Exchanger heat-transfer surface, sq m
Am	=	Coefficient of the mth value dimensionless
Ao	8	Coefficient, dimensionless
A1	Ż	1st order coefficient to be multiplied by N _{tu} , dimensionless
A2	-	2nd order coefficient to be multiplied by N_{tu}^2 , dimensionless
A3	=	3rd order coefficient to be multiplied by N_{tu}^3 , dimensionless
A4	2	4th order coefficient to be multiplied by N_{tu}^4 , dimensionless
A5	3	5th order coefficient to be multiplied by N _{tu} ⁵ , dimensionless
a	=	Exchanger heat-transfer surface, sq m/m
С		Capacity rate, W°/K. Also designates dimensionless arbitrary constant
Cpc	Ŧ	Specific heat at constant pressure of cold fluid, J/kg°K
Cph	=	Specific heat at constant pressure of hot fluid, J/kg°K
D	=	Empirical value of effectiveness (computer generated), dimensionless
е	=	Error. Also used as the exponential function
F	2	Logarithmic mean temperature difference correction factor, dimensionless
L	Ŧ	Exchanger length, m
N	=	Number of effectiveness empirical data points used to determine a curve for R, dimensionless

n	=	Number of tube passes, dimensionless. Also number of equations, dimensionless
n _c	=	A related N_{tu} per unit length hot side, m ⁻¹
nh	=	A related Nt_u per unit length cold side, m ⁻¹
N _{tu}	=	Number of transfer units, dimensionless
Р	×	Temperature group, dimensionless
q	=	Total rate of heat transfer, W
qmaz	τ=	Maximum total rate of heat transfer, W
R	=	Capacity rate ratio, dimensionless
S	=	Temperature group, dimensionless
sr	=	Sum of the squares of the residuals, dimensionless
т	=	Hot fluid temperature, °C
Трі	=	Particular integral, dimensionless
T1	=	Hot fluid temperature in, °C
T2	æ	Hot fluid temperature out, *C
t1	=	Cold fluid temperature in, °C
t2	=	Cold fluid temperature out, °C
ta	=	Cold fluid temperature 1st pass, °C
tab	z	Cold fluid temperature between 1st and 2nd passes
tb	=	Cold fluid temperature 2nd pass, °C
tbc	æ	Cold fluid temperature between 2nd and 3rd passes
tc	=	Cold fluid temperature 3rd pass, °C
U	H	Overall heat transfer coefficient, $W/m^2 - C$
W	=	Mass flow, kg/sec. Also the product of ω and L, dimensionless
-	=	length coordinate m. Also used to represent

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= length coordinate, m. Also used to represent a constant value in a sequence

<pre>y = Sum of mth degree polynomial, defined by eq. (53), dimensionless</pre>
Z = A product of z and L, dimensionless
z = A related N _{tu} per unit length, hot side, 1/m
Greek Letter Symbols
α = Root of auxiliary differential equation, 1/m
ε = Exchanger effectiveness, dimensionless
λ = Combination of variables defined by equation (11), dimensionless
ω = A related N _{tu} per unit length, hot side, 1/m
<pre></pre>
Σ = Summation, dimensionless
σ^2 = Variance, dimensionless
$\theta_{\rm m}$ = Mean temperature difference for exchanger, °C
<pre>a = Indicates partial derivative, dimensionless</pre>
Subscripts
c = Cold fluid
h = Hot fluid
i,j,k = Values in a sequence

4 N 4 8 4 N 4 8 8 8 8

T.T.T. T. T.

- m = Degree or order, an exponent
- 1 = inlet

2 = outlet

Special Symbols

[A] = An m x n matrix, symmetric

- [K] = An m x n matrix, symmetric
- [L] = A lower triangular matrix
- [T] = An m x 1 vector
- [0] = A null vector

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I. INTRODUCTION

A. BACKGROUND

When analyzing the standard counterflow heat exchanger, it becomes apparent that, from a practical standpoint, it is often difficult to obtain a high velocity for one of the fluids when this fluid is constrained to flow through all of the tubes in a single pass. This leads to a possibility of a low overall heat transfer coefficient which cancels the advantage of the high logarithmic mean temperature difference which is obtainable in true counterflow.

The quest for flow arrangements for increased heat recovery has led to arrangements that yield increased tubeside velocities and higher overall heat transfer coefficients even at the expense of a departure from the ideal true counterflow arrangement. Thus, the design may be modified so that the tube side fluid is carried through fractions of the tubes consecutively.

Heat exchangers of this type with one shell pass and n tube passes are often referred to as 1-n exchangers. These exchangers, such as the one shell pass, two-tube pass (1-2) parallel-counterflow exchanger (see Figure 1.1), are configured such that all of the tube side fluid flows through the two halves of the the tubes successively. A single channel is employed with a partition to permit the



Figure 1.1 1-2 Parallel-Counterflow Exchanger

entry and exit of the tube side fluid from the same channel. Note the baffles used to induce turbulence causing the liquid to flow through the shell at right angles to the axes of the tubes thus helping to create a higher shell side velocity with higher shell side heat-transfer coefficients.

To date, much work has been done on finding the true logarithmic mean temperature difference for heat exchangers with an even number of tube passes. Little work, however, has been done with regard to heat exchangers having an odd number of passes. This is primarily due to the method employed in deriving an analytical solution to measure the overall effectiveness of a heat exchanger.

Exhangers with an even number of tube passes often present a configuration problem especially in a marine application where the inlet and outlet of the cooling fluid must

be one the same side of the exchanger header (see Figure 1.1). This particular problem could be alleviated by going to heat exchangers with an odd number of tube passes. Currently, precise mathematical expressions for the effectiveness of the 1-3 exchangers do not exist. Hence, a theoretical examination is reported on here which considers the effectiveness of the 1-3 parallel-counter flow exchanger which is shown in Figure 1.2.

Before doing this it is important to mention the work that lead to the effectiveness method and the development of work on heat exchangers with an odd number of tube passes.



Figure 1.2 1-3 One Shell Pass Three Tube Pass Exchanger

This will be considered in Section II. Kern [Ref. 1: pp. 224-226], makes the interesting point that the optimum exchanger requires an exchanger capable of providing the optimum fluid-flow velocities on the shell as well as the tube sides. This might frequently entail the use of an odd number of tube passes or an odd tube length.

B. WHY EFFECTIVENESS AS A FUNCTION OF N_{tu}

It is also noted that Kays and London [Ref. 2: pp. 24-29] indicated that the effectiveness as a function of N_{tu} ($\epsilon - N_{tu}$) method is the favored approach for evaluating a heat exchanger's performance because:

- 1. The effectiveness value stands alone as a dependent variable and should not appear directly in the abscissa and indirectly in the ordinate of a graphical display.
- 2. The log-mean difference equation misleadingly simplifies the notion of what is involved in heat exchanger design theory, since the implication is that only a rate equation is required.
- 3. The ϵ N_{tu} approach simplifies the algebra involved in predicting the performance of complex flow arrangements.
- 4. The more meaningful arguments are related to ease of use in design work. Two prime examples of these are:
 - a) Given the overal heat transfer coefficient, U, the two fluid capacity rates, C_C and C_h , and the terminal temperatures, determine the required surface area, A.
 - b) Given A, U, C_c , C_h and the inlet temperatures of both streams, determine the outlet temperatures.

II. THE DEVELOPMENT OF THE EFFECTIVENESS METHOD

A. LITERATURE SURVEY

It was Nagle [Ref. 3: pp. 604-609], in 1931, who credited Davis [Ref. 4] with a simplified method for computing actual temperature differences between two heatinterchanging streams which depart from true counter or concurrent (parallel) flow. This is now the familiar "F factor" method which expresses the actual mean temperature difference θ_m in $q = UA \theta_m$ as a fraction F of the counterflow logarithmic mean temperature difference, LTMD, θ_{mc} via $\theta_m = F \theta_{mc}$.

The example of initial interest was the 1-2 exchanger with a single shell pass and two continuous tube passes in counter and concurrent flow with it. The method involved derivation of the actual temperature difference for the flow pattern and formed the ratio $F = \theta_m/\theta_{mc}$. This familiar LMTD correction factor was plotted conveniently as functions of the effectiveness, ε , and the capacity rate ratio R with R as a parameter. These mean temperature difference correction charts are available for many flow arrangements [Ref. 1: pp. 829-833 and Ref. 5]. The effectiveness, ε (often called P or S), is always the cold fluid effectiveness and R is always the capacity rate ratio of cold fluid to hot fluid.

Nagle detailed assumptions and derivations for the 1-2, 1-4 and 1-6 exchangers. The F factors were obtained by Nagle through graphical integration and were accompanied by the comment that F factors for the 1-2 exchanger could be applied with negligible error to 1-4 and 1-6 exchangers. Underwood [Ref. 6: pp. 145-148] rederived the equations of Nagle for 1-2 and 1-4 exchangers to eliminate the need for obtaining F factors by graphical integration.

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Bowman [Ref. 7: pp. 541-544] pointed out that for a very large or infinite number of tube passes, the F factor approached, as a limit, its value in crossflow with both fluids completely mixed. It was further stated that even at the limit, the F factors were only 1 to 2 percent lower than those for the 1-2 exchanger. A previous paper by Kraus and Kern [Ref. 8] did not confirm the generalization that 1-n exchangers differed only negligibly from the 1-2 exchanger although this lack of confirmation was obtained on an $\varepsilon = f(R, N_{tu})$ basis. Moreover, the Kraus-Kern work does not confirm the generalizations on an $F = f(R, N_{tu}, \varepsilon)$ basis.

From the standpoint of usefulness and good accuracy, it is essential that F factors, if they are to be used in preference to $\varepsilon = f(R, N_{tu}, flow arrangement)$, be obtained with precision. Plots of F = f(R, ε , = P or S) [Ref. 1: pp. 829-833 and Ref. 5] show that the curves for particular values of R approach infinite slope as F decreases. While this can be partially alleviated by restricting R < 1.0 (a

constraint used in the $\varepsilon = f(R, N_{tu}, flow arrangement)$ approach), it is seen that small errors in the interpolation for R or $\varepsilon = P$ or S can result in large fluctuations in the value of F.

In a comprehensive paper, Bowman, Mueller and Nagle [Ref. 9: pp. 283-294] presented graphs of F factors for shells with one through six shell passes and numbers of continuous tube passes respectively double the number of shell passes. In view of the earlier references to Nagle and Bowman, it should be noted that F factors were computed for the 1-2 exchanger in [Ref. 9: pp. 283-294] using the equations of Underwood [Ref. 6: pp. 145-158].

Ten Broeck [Ref. 10: pp. 1041-1042] prepared a graph of the dimensionless groups now known as ε , R and N_{tu} for the 1-2 exchanger. Such a graph had the added versatility of simplifying the calculation of performance in a given exchanger when operating at conditions different for those for which it was designed. Kays and London [Ref. 2: pp. 63-74] prepared similar graphs and tables of $\varepsilon = f(R, N_{tu}, flow arrangement)$ for the 1-2 exchanger and for several cases of crossflow and periodic flow.

The foregoing describes the early history of the search for the so-called Logarithmic Mean Temperature Difference Correction Factor, F, with regard to heat exchangers having an even number of tube passes. It is a fact, however, that certain space economies could be realized from exchangers

having an odd number of tube passes so that the tube side fluid could enter and leave the exchanger at opposite ends of the exchanger (see Figure 1.2).

B. FISCHER'S WORK

With the foregoing in mind, an extensive search has been conducted to obtain $\varepsilon - N_{tu}$ data for the so called "1-3" and "1-5" exchangers. This search has uncovered a single work, that of Fischer [Ref. 11: pp. 377-383], which summarizes the historical development covered here and contains only a small section on the 1-3 exchanger. This work by Fischer develops an equation for true mean temperature difference of the 1-3 exchanger and casts the results in terms of F rather than ε . Moreover, the work treats only the case where the three tube passes are arranged with two in counterflow and one in parallel flow (1-3:2C) making no mention of the one counterflow and two parallel flow (1-3:2P) case (see Figures 2.1 and 2.2). In addition, the equation developed to yield F must be solved using a trial and error solution.

The present work is aimed at continuing the Fischer investigation for several reasons:

- 1. A solution is needed for effectiveness, ε , as a function of capacity rate (R) and number of transfer units (N_{tu}).
- 2. This solution should be in a closed form if at all possible so that it will be computationally efficient and useful in both the design and analysis frameworks.



Figure 2.1 1-3:2C Three Tube Passes - Two in Counterflow





- 3. Because valid design data can evolve from a polynomial approximation. The search should not be abandoned just because a closed form solution does not result from an analytical approach.
- 4. Data is needed for the (1-3:2P) two parallel-one counterflow configuration.

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5. A 1-3 exchanger in a marine (shipboard) application may result in a considerable space saving over its 1-n counterpart with n even. This would be evident on the outside of the exchanger where it would be immediately noted that the 1-3 exchanger has tube side inlet and outlet at opposite ends of the exchanger.

The next section confirms Fischer's result and shows that a closed form solution cannot be obtained for the effectiveness of the 1-3 exchanger. Sections IV and V demonstate how, through numerical analysis assisted by a computer, a polynomial solution can be derived that will yield the effectiveness to engineering accuracy.

III. AN ATTEMPT AT A CLOSED FORM SOLUTION

A. EFFECTIVENESS AS A FUNCTION OF CAPACITY RATES AND EXCHANGER SIZE

This section deals with an investigation into the effectiveness, ε , of a one shell pass and three tube pass heat exchanger, whereby ε compares the actual heat transfer rate to the thermodynamically limited, maximum possible heat transfer rate as would be realized only in a counter flow heat exchanger of infinite transfer area. This exchanger heat transfer effectiveness is given by

$$\varepsilon = \frac{q}{q_{\max}} = \frac{C_h(T_{hot,in} - T_{hot,out})}{C_{\min}(T_{hot,in} - t_{cold,in})} = \frac{C_c(t_{cold,out} - t_{cold,in})}{C_{\min}(T_{hot,in} - t_{cold,in})}$$

where C_{\min} is the smaller of the C_h and C_c magnitudes. Thus, ε possesses the significance of effectiveness of the heat exchanger from a thermodynamic point of view, with the magnitude of the effectiveness completely defining the heat transfer performance. In general we express $\varepsilon = f(N_{tu}, R,$ and flow arrangement) and when the flow arrangement is understood, it is said that $\varepsilon = f(N_{tu}, R)$. [Ref. 2: pp. 14-26].

The number of heat transfer units N_{tu} is a nondimensional expression of the "heat transfer size" of the exchanger. When N_{tu} is small the exchanger effectiveness is low, and when N_{tu} is large, ε approaches the limit imposed by the flow

arrangement and thermodynamic conditions asymptotically. From inspection of the definition of N_{tu}

$$N_{tu} = \frac{AU}{C_{min}} = \frac{1}{C_{min}} \int_{\Omega}^{A} U dA$$

it is clear that the overall conductance and transfer area affect the costs of attaining a high value for N_{tu} , ergo high ε . The capacity rate ratio, R, as defined by

$$R = \frac{C_{\min}}{C_{\max}}$$

is simply the ratio of mass flow rate times specific heat capacity for the two streams. These can be considered as flow stream thermal-capacity rates, i.e., energy storage rate in the stream per unit of temperature change. [Ref. 2: pp. 14-26]

The attempt taken in this thesis to develop a closed form solution has used the basic fundamentals of heat transfer as well as those indicated above. A closed form solution for ε was sought for both 1-3 exchangers with one having two out of three tube passes in parallel flow and the other having two out of three tube passes in counterflow. The analytical approach taken, and demonstrated in this section, is for two out of three tube passes in counterflow.

B. ANALYTICAL DEVELOPMENT

The derivation for the effectiveness, ε , of the 1-3 exchanger as a function of the capacity rate ratio, R, and number of transfer units, N_{tu}, depends on several assumptions.

- (1) The overall coefficient of heat transfer, U, does not vary within the exchanger.
- (2) The specific heat of both hot side and cold side fluids does not vary.
- (3) Each fluid is thoroughly mixed, that is, the temperature of both hot and cold side fluids is uniform over any cross section.
- (4) Steady flow conditions are maintained.
- (5) Heat losses to or from the environment are negligible.
- (6) No change of phase takes place; all heat transferred is sensible heat.
- (7) There is equal heat transfer surface in each pass.

The configuration is shown in Figure 3.1 where the three tube passes are designated with subscripts a, b and c. The temperature of the hot (shell side) fluid is indicated by upper case letters. For the cold (tube side) fluid, lower case letters are used. The subscript 1 always refers to the fluid inlet and the subscript 2 always refers to the fluid outlet.

With W_h and C_{ph} designating mass flow (kg/sec) and specific heat (Joules/kg^{-•}K) of hot fluid entering at T_1 and leaving at T_2 we define a capacity rate for the hot side

 $C_h = W_h C_{ph}$





In similar fashion for the cold side (with W_C and C_{pc}) entering at t₁ and leaving at t₂, we have

$$C_c = W_c C_{pc}$$

.....

We then obtain an energy balance for the entire exchanger

$$C_h(T_1 - T_2) = C_c(t_2 - t_1)$$
 (1)

Over the right hand side of the exchanger (Figure 3.1)

$$C_h(T_1 - T) = C_c(t_2 - t_c + t_b - t_a)$$
 (2)

and a differentiation gives

$$C_{h}dT = C_{c}(dt_{c} - dt_{b} + dt_{a})$$
(3)

Across dx, with a (m^2/m) , the surface per running meter of length of pass so that A = 3aL is the total surface in the exchanger, we may write the heat transferred to the element dx in each cold pass.

 $C_{c}dt_{a} = Uadx(T - t_{a})$ (4a)

$$C_{c}dt_{b} = -Uadx(T - t_{b})$$
(4b)

$$C_{c}dt_{c} = Uadx(T - t_{c})$$
(4c)

Here it should be observed that due cognizance has been taken of the direction of the flow in each cold fluid pass with respect to the positive sense of the length coordinate, x, and U is the overall heat transfer coefficient $(W/m^{2-\circ}C)$. With eqs. (4) in eq. (3)

$$C_{n}dT = Ua(3T - t_{n} - t_{n})dx$$

or

$$\frac{dT}{dx} = n_{h}(3T - t_{a} - t_{b} - t_{c})$$
(5)

where

$$n_h = \frac{Ua}{C_h}$$

is a sort of N_{tu} per unit length for the hot side. Now differentiate eq. (5)

$$\frac{d^2 T}{dx^2} = n_h (3\frac{dT}{dx} - \frac{dt_a}{dx} - \frac{dt_b}{dx} - \frac{dt_c}{dx})$$

and with eqs. (4) substituted

$$\frac{d^2T}{dx^2} = 3n_h \frac{dT}{dx} - n_c n_h (T - t_a + t_b - t_c)$$
(6)

where

$$n_c = \frac{Ua}{C_c}$$

where again the resemblence of n_c to N_{tu} can be noted.

From eq. (2) we obtain

$$\frac{C_{h}}{C_{c}} (T_{1} - T) - t_{2} = t_{b} - t_{a} - t_{c}$$
(7)

and with eq. (7) put into eq. (6) we obtain

$$\frac{d^2T}{dx^2} - 3n_h \frac{dT}{dx} = -n_c n_h [T + \frac{C_h}{C_c} (T_1 - T) - T_2]$$

or

$$\frac{d^2 T}{dx^2} - 3n_h \frac{dT}{dx} = -n_c n_h \frac{C_h}{C_c} [(R_c - 1)T + T_1 - R_c t_2]$$
(8)

where

$$R_c = C_c/C_h$$

is the capacity rate ratio for the cold side.

Notice that

$$n_c n_h \frac{C_h}{C_c} = \frac{Ua}{C_c} \cdot \frac{Ua}{C_h} \cdot \frac{C_h}{C_c} = \left(\frac{Ua}{C_c}\right)^2 = m$$

and

$$R_{h} = \frac{1}{R_{c}} = \frac{C_{h}}{C_{c}}$$

a capacity rate ratio for the hot side. Then, algebraic adjustment provides

$$\frac{d^2 T}{dx^2} - 3n_h \frac{dT}{dx} + m \left(\frac{1 - R_h}{R_h}\right)T = m(\frac{t}{R_h} - T_1)$$
(9)

-

which is a linear, non-homogeneous, second order differential equation with constant coefficients having a complementary function

$$T_{c} = C_{1}e^{\alpha_{1}x} + C_{2}e^{\alpha_{2}x}$$
(10)

where C_1 and C_2 are arbitrary constants and where

$$\alpha_1, \alpha_2 = \frac{3n_h}{2} \pm \frac{1}{2} [9n_h^2 - 4m(\frac{1 - R_h}{R_h})]^{1/2}$$

$$= \frac{3n_{h}}{2} \pm \frac{n_{h}}{2} \left[9 - \frac{4m}{n_{h}^{2}} \left(\frac{1 - R_{h}}{R_{h}}\right)\right]^{1/2}$$

But

$$\frac{m}{n_{h}^{2}} = \frac{(Ua)^{2}}{(C_{c})^{2}} \cdot \frac{(C_{h})^{2}}{(Ua)^{2}} = (\frac{C_{h}}{C_{c}})^{2} = R_{h}^{2} = \frac{1}{R_{c}^{2}}$$

so that

$$\alpha_1, \alpha_2 = \frac{3n_h}{2} \pm \frac{n_h}{2} [9 - 4R_h^2 (\frac{1 - R_h}{R_h})]^{1/2}$$

or

$$\alpha_1, \alpha_2 = \frac{n_h}{2} (3 \pm \lambda)$$
(11)
where

$$\lambda = [9 - 4R_{\rm h}(1 - R_{\rm h})]^{1/2}$$
(12)

Designate the particular integral as T_{pi} and by the method of undetermined coefficients let $T_{pi} = P$ so that in eq. (9)

$$m(\frac{1 - R_h}{R_h}) P = m(\frac{t_2}{R_h} - T_1)$$

This makes

$$T_{pi} = P = [\frac{t_2}{R_h} - T_1] [\frac{R_h}{1 - R_h}]$$

so that

$$^{T}pi = \frac{t_{2} - R_{h}T_{1}}{1 - R_{h}}$$
(13)

The general solution to eq. (9) is the sum of eqs. (10) and (13)

$$T(x) = C_1 e^{\alpha_1 x} + C_2 e^{\alpha_2 x} + \frac{t_2 - R_h T_1}{1 - R_h}$$
(14)

where the arbitary constants, C_1 and C_2 are evaluated from conditions at x = 0 and x = L. At x = 0, $T(x = 0) = T_2$ and at x = L, $T(x = L) = T_1$. When these are inserted, in turn, into eq. (14), one obtains a pair of linear algebraic equations in the unknowns C_1 and C_2

$$T_2 = C_1 + C_2 + T_{pi}$$

 $T_1 = C_1 e^{\alpha_1 L} + C_2 e^{\alpha_2 L} + T_{pi}$

where T_{pi} is given by eq. (13).

It is only a matter of algebra to show that

$$C_{1} = \frac{(T_{1} - T_{pi}) - (T_{2} - T_{pi})e^{\alpha_{2}L}}{e^{\alpha_{1}L} - e^{\alpha_{2}L}}$$
(15a)

and

$$C_{2} = \frac{(T_{2} - T_{pi})e^{\alpha_{1}L} - (T_{1} - T_{pi})}{e^{\alpha_{1}L} - e^{\alpha_{2}L}}$$
(15b)

It is easy to see from eq. (1) that

$$R_{h} = \frac{C_{h}}{C_{c}} = \frac{(t_{2} - t_{1})}{(T_{1} - T_{2})}$$

so that

$$t_2 = t_1 + R_h(T_1 - T_2)$$

Use of this in eq. (13) shows that

$$T_{pi} = \frac{t_1 + R_h(T_1 - T_2) - R_h T_1}{1 - R_h}$$

or

$$\Gamma_{\rm pi} = \frac{t_1 - R_{\rm h} T_2}{1 - R_{\rm h}}$$
(16)

indicating two alternative forms for T_{pi} given by eqs. (13) and (16).

Insertion of eqs. (13) and (16) in eqs. (15) for C_1 and C_2 will yield after some algebra

$$C_{1} = \frac{(\frac{T_{1} - t_{2}}{1 - R_{h}}) - (\frac{T_{2} - t_{1}}{1 - R_{h}}) e^{\alpha_{2}L}}{e^{\alpha_{1}L} - e^{\alpha_{2}L}}$$
(17a)

and

$$C_{2} = \frac{\begin{pmatrix} T_{2} - t_{1} \\ 1 - R_{h} \end{pmatrix} e^{\alpha_{1}L} - \begin{pmatrix} T_{1} - t_{2} \\ 1 - R_{h} \end{pmatrix}}{e^{\alpha_{1}L} - e^{\alpha_{2}L}}$$
(17b)

Equation (14) is an expression for the hot side temperature at any location in the exchanger in terms of the extreme temperatures, t_1 , t_2 , T_1 and T_2 .

Next take eq. (5) and set it equal to the derivative of eq. (14) noting that C_1 , C_2 and T_{pi} are all known constants.

$$\frac{dT}{dx} = n_{h}(3T - t_{a} - t_{b} - t_{c}) = \alpha_{1}C_{1}e^{\alpha_{1}x} + \alpha_{2}C_{2}e^{\alpha_{2}x}$$
(18)

At x = 0, where $T = T_2$, $t_a = t_1$ and $t_b = t_c = t_{bc}$

$$\frac{dT}{dx} = n_h (3T_2 - t_1 - 2t_{bc}) = \alpha_1 C_1 + \alpha_2 C_2$$
(19)

and if we subtract eq. (4a) from eq. (4c) we obtain

$$\frac{dt_a - dt_c}{t_a - t_c} = -\frac{Ua}{C_c} dx = -n_c dx$$

which can be integrated using C_3 as the constant of integration.

$$t_a - t_c = C_3 e^{-n_c x}$$

and at x = 0 where $t_a = t_1$ and $t_c = t_{bc}$

$$t_1 - t_{bc} = C_3$$

or

 $t_{bc} = t_1 - C_3$

In addition at x = L, $t_a = t_{ab}$ and $t_c = t_2$ so that

$$t_{ab} - t_2 = C_3 e^{n_c L}$$

or

$$C_3 = \frac{t_{ab} - t_2}{-n_h L} = (t_{ah} - t_2)e^{n_c L}$$

This gives a relationship between ${\tt t}_{ab}$ and ${\tt t}_{bc}$

$$t_{bc} = t_1 - (t_{ab} - t_2)e^{N_c}$$
 (20)

where $N_C = n_C L$ can be considered as the total number of transfer units for the cold side.

Return now to eq. (18) and look at the conditions at x = L where $t_a = t_b = t_{ab}$, $t_c = t_2$ and $T = T_1$. These conditions in eq. (18) give

$$n_{h}(3T_{1} - 2t_{ab} - t_{2}) = \alpha_{1}C_{1}e^{\alpha_{1}L} + \alpha_{2}C_{2}e^{\alpha_{2}L}$$

where again we remember that C_1 and C_2 are known constants. Solving for t_{ab}

$$2t_{ab} = -\frac{1}{n_b} (\alpha_1 C_1 e^{\alpha_1 L} + \alpha_2 C_2 e^{\alpha_2 L}) + 3T_1 - t_2$$

and with this in eq. (20)

$$2t_{bc} = e^{N_{c}} \left[\frac{1}{n_{b}} \left(\alpha_{1}C_{1}e^{\alpha_{1}L} + \alpha_{2}C_{2}e^{\alpha_{2}L} \right) + 3t_{2} - 3T_{1} \right] + 2t_{1}$$

Then with eq. (21) in eq. (19)

$$\alpha_{1}C_{1} + \alpha_{2}C_{2} = 3n_{h} \left[(T_{2} - t_{1}) + e^{N_{c}}(T_{1} - t_{2}) \right]$$
$$-e^{N_{c}}(\alpha_{1}C_{1}e^{\alpha_{1}L} + \alpha_{2}C_{2}e^{\alpha_{2}L}) \qquad (22)$$

Equation (22) confirms Fischer's result [Ref. 12: pp. 377-383] and at this point in his development he branches off to seek an expression for the Logarithmic Mean Temperature Difference Correction Factor, F. The attention here is focused on $\varepsilon = f(R, N_{tu})$ and the balance of this section continues in this vein.

Look at the $\alpha_1C_1 + \alpha_2C_2$ term in eq. (22). Use of eq. (11) allows the representation of $\alpha_1C_1 + \alpha_2C_2$

$$\frac{n_{h}}{2} (3 + \lambda)C_{1} + \frac{n_{h}}{2} (3 - \lambda)C_{2}$$
(23a)

or

$$\frac{3n_{h}}{2} (C_{1} + C_{2}) + \frac{\lambda n_{h}}{2} (C_{1} - C_{2})$$
(23b)

Then from eqs. (17)

$$C_{1} + C_{2} = \frac{(T_{2} - t_{1})[e^{\alpha_{1}L} - e^{\alpha_{2}L}]}{(1 - R_{h})[e^{\alpha_{1}L} - e^{\alpha_{2}L}]}$$

or

$$C_1 + C_2 = \frac{T_2 - t_1}{1 - R_h}$$
 (24)

Moreover

$$C_{1} - C_{2} = \frac{2(T_{1} - t_{2}) - (T_{2} - t_{1})[e^{\alpha_{1}L} + e^{\alpha_{2}L}]}{(1 - R_{h})[e^{\alpha_{1}L} - e^{\alpha_{2}L}]}$$
(25)

Now let

$$\omega = \frac{3n_{\rm h}}{2} \tag{26a}$$

and

$$z = \frac{\lambda n_h}{2}$$
(26b)

so that

 $e^{\alpha_1 L} - e^{\alpha_2 L} = e^{(\omega + z)L} - e^{(\omega - z)L}$ $= e^{\omega L} e^{zL} - e^{\omega L} e^{-zL}$ $= e^{\omega L} (e^{zL} - e^{-zL})$

or

$$e^{\alpha_1 L} - e^{\alpha_2 L} = 2e^{\omega L} \sinh zL$$
 (27)

Moreover, it is easy to see that

$$e^{\alpha_1 L} + e^{\alpha_2 L} = 2e^{\omega L} \cosh zL \qquad (28)$$

If eqs. (24) through (28) are collected and put into the expression of eq. (23b), the result is

$$\alpha_1 C_1 + \alpha_2 C_2 = \frac{3n_h}{2} (C_1 + C_2) + \frac{\lambda n_h}{2} (C_1 - C_2)$$
 (23b)

$$= \omega \left[\frac{T_2 - t_1}{1 - R_h} \right] + z \left[\frac{2(T_1 - t_2) - (T_2 - t_1)2e^{\omega L} \operatorname{csch} zL}{(1 - R_h)2e^{\omega L} \operatorname{sinh} zL} \right]$$

$$\alpha_1 C_1 + \alpha_2 C_2 = \omega(\frac{T_2 - t_1}{1 - R_h}) +$$

$$z[(\frac{T_1 - t_2}{1 - R_h})e^{-\omega L} \operatorname{csch} zL - (\frac{T_2 - t_1}{1 - R_h}) \operatorname{coth} zL] \quad (29a)$$

and this could also be written as

$$\alpha_1 C_1 + \alpha_2 C_2 = (\frac{T_2 - t_1}{1 - R_h}) [\omega - z \text{ coth } zL]$$

+
$$z(\frac{T_1 - t_2}{1 - R_h})e^{-\omega L} csch zL$$
 (29b)

The next step is to reduce the right hand side of eq. (22). Use of eq. (11) permits the representation

$$a_1C_1e^{a_1L} + a_2C_2e^{a_2L} = (\frac{3n_h}{2} + \frac{\lambda n_h}{2})C_1e^{a_1L} + (\frac{3n_h}{2} - \frac{\lambda n_h}{2})C_2e^{a_1L}$$

or with eqs. (26) inserted

$$\alpha_{1}C_{1}e^{\alpha_{1}L} + \alpha_{2}C_{2}e^{\alpha_{2}L} = \omega(C_{1}e^{\alpha_{1}L} + C_{2}e^{\alpha_{2}L}) + z(C_{1}e^{\alpha_{1}L} - C_{2}e^{\alpha_{2}L})$$
(30)

But by eqs. (11) and (17)

$$C_{1}e^{\alpha_{1}L} + C_{2}e^{\alpha_{2}L} = \frac{(T_{1}-t_{2})(e^{\alpha_{1}L} - e^{\alpha_{2}L}) + (T_{2}-t_{1})[e^{(\alpha_{1}+\alpha_{2})L} - e^{(\alpha_{1}+\alpha_{2})L}]}{(1 - R_{h})(e^{\alpha_{1}L} - e^{\alpha_{2}L})}$$

or

or

$$C_1 e^{\alpha_1 L} + C_2 e^{\alpha_2 L} = \frac{T_1 - t_2}{1 - R_h}$$
 (31)

 $C_1 e^{\alpha_1 L} - C_2 e^{\alpha_2 L}$ may also be simplified. Again using

eqs. (11), (17a) and (17b)

$$C_{1}e^{\alpha_{1}L} - C_{2}e^{\alpha_{2}L} = \frac{(T_{1}-t_{2})(e^{\alpha_{1}L} + e^{\alpha_{2}L}) - (T_{2}-t_{1})[e^{(\alpha_{1}+\alpha_{2})L} + e^{(\alpha_{1}+\alpha_{2})L}]}{(1 - R_{h})(e^{\alpha_{1}L} - e^{\alpha_{2}L})}$$

The exponential term at the far right in the numerator is really quite simple. From eq. (11)

 $\alpha_1 + \alpha_2 = (\frac{3n_h}{2} + \frac{\lambda n_h}{2}) + (\frac{3n_h}{2} - \frac{\lambda n_h}{2}) = 3n_h$

and by eq. (26a) $\alpha_1 + \alpha_2 = 3n_h = 2\omega$. Thus with the combination of exponentials in the numerator and the denominator given by eqs. (27) and (28) we find that

$$C_1 e^{\alpha_1 L} - C_2 e^{\alpha_2 L} = \frac{2(T_1 - t_2)e^{\omega L} \cosh zL - (T_2 - t_1)2e^{2\omega L}}{2(1 - R_h)e^{\omega L} \sinh zL}$$

or

$$C_1 e^{\alpha_1 L} - C_2 e^{\alpha_2 L} = (\frac{T_1 - t_2}{1 - R_h}) \operatorname{coth} zL - (\frac{T_2 - t_1}{1 - R_h}) e^{\omega L} \operatorname{csch} zL (32)$$

Now with eqs. (31) and (32) put into eq. (30) we obtain

$$\alpha_1 C_1 e^{\alpha_1 L} + \alpha_2 C_2 e^{\alpha_2 L} = \omega(\frac{T_1 - T_2}{1 - R_h})$$

or

+
$$z[(\frac{T_1 - t_2}{1 - R_h}) \operatorname{coth} zL - (\frac{T_2 - t_1}{1 - R_h})e^{\omega L} \operatorname{csch} zL]$$

$$\alpha_{1}C_{1}e^{\alpha_{1}L} + \alpha_{2}C_{2}e^{\alpha_{2}L} = (\frac{T_{1} - t_{2}}{1 - R_{h}}) [\omega + z \text{ coth } zL]$$
$$- z(\frac{T_{2} - t_{1}}{1 - R_{h}})e^{\omega L} \operatorname{csch} zL \qquad (33)$$

With eqs. (29b) and (33) inserted into eq. (22) we obtain

$$\left(\frac{T_2 - t_1}{1 - R_h}\right) \left[\omega - z \operatorname{coth} zL\right] + z\left(\frac{T_1 - t_2}{1 - R_h}\right)e^{-\omega L}\operatorname{csch} zL =$$

$$3n_h[(T_2 - t_1) + e^{N_c}(T_1 - t_2) -$$

$$e^{-N_{c}}\left(\frac{T_{1}-t_{2}}{1-R_{h}}\right)(\omega + z \operatorname{coth} zL) - z\left(\frac{T_{2}-t_{1}}{1-R_{h}}\right)e^{\omega L}\operatorname{csch} zL\right]$$
(34)

We wish to develop an expression for the exchanger effectiveness so we designate the hot side effectiveness as

$$\epsilon_{\rm h} = \frac{{\rm T}_1 - {\rm T}_2}{{\rm T}_1 - {\rm t}_1}$$
 (35)

and we begin by simplifying eq. (34) by dividing throughout by $(T_2 - t_1)/(1 - R_h)$ to obtain

$$(\omega - z \operatorname{coth} Z) + ze^{-W} \operatorname{csch} z(\frac{T_1 - t_2}{T_2 - t_1}) = R + e^{N_c} R(\frac{T_1 - t_2}{T_2 - t_1})$$

$$-e^{-N}c[(\omega + z \operatorname{coth} Z) (\frac{T_1 - t_2}{T_2 - t_1}) - ze^{\Psi} \operatorname{csch} Z] \quad (36)$$

where

$$W = \omega L \tag{37a}$$

$$Z = zL \tag{37b}$$

and

$$R = 3n_h(1 - R_h)$$
 (37c)

We can then let

$$\phi = \frac{T_1 - t_2}{T_2 - t_1}$$

and get eq. (36) to look like

$$(\omega - z \operatorname{coth} Z) + (ze^{-W} \operatorname{csch} Z) \phi =$$

$$R + (e^{N} R)\phi - [e^{-N} c(\omega + \coth Z)]\phi - (ze^{(W-N} c) csch Z)$$

It is now a matter of algebra to solve for ϕ

$$\phi = \frac{R + ze}{ze^{-W} \operatorname{csch} Z - (\omega - z \operatorname{coth} Z)}$$
(38)
$$\frac{N}{ze^{-W} \operatorname{csch} Z - \operatorname{Re}^{C} + e^{-C}(\omega + z \operatorname{coth} Z)}$$

The next step is to represent ϕ as a function of ε_h . This is done with some algebraic symmastics as follows:

$$\phi = \frac{T_1 - t_2}{T_2 - t_1} = \frac{T_1 - t_2}{T_2 - T_1 + T_1 - t_1} = \frac{T_1 - t_2}{(T_1 - t_1)! \frac{T_1 - T_2}{T_1 - t_1}}$$

or

$$\phi = \frac{T_1 - t_2}{(T_1 - t_1)(1 - \epsilon_h)}$$

Moreover

$$\phi = \frac{T_1 - t_2}{(T_1 - t_1)(1 - \epsilon_h)} = \frac{T_1 - t_1 + t_1 - t_2}{(T_1 - t_1)(1 - \epsilon_h)}$$

$$= \frac{(T_1 - t_1) [1 - (\frac{t_2 - t_1}{T_1 - t_1}) (\frac{T_1 - T_2}{T_1 - T_2})]}{(T_1 - t_1)(1 - \epsilon_h)}$$

$$= \frac{1 - (\frac{T_1 - T_2}{T_1 - t_1}) (\frac{t_2 - t_1}{T_1 - T_2})}{1 - \epsilon_h}$$

But $R_h = (t_2 - t_1)/(T_1 - T_2)$ so that

$$\phi = \frac{1 - \epsilon_h R_h}{1 - \epsilon_h}$$

or

$$\epsilon_{\rm h} = \frac{\phi - 1}{\phi - R_{\rm h}} \tag{39}$$

where ϕ is given by eq. (38)

The neatness of the form of eq. (39) is deceptive because, unfortunately, it cannot be used to determine a unique value for ε_h . The reason for this can be found in an inspection of eq. (38) which provides the value of ϕ which is used in eq. (39).

Notice in eq. (38) that 7, W and N_C are all functions of the product al. On the other hand, w, z and R are functions of a only. Thus, it is impossible to vary a and L independently and still achieve a unique solution.

For example, suppose $a = 50 \text{ m}^2/\text{m}$ and L = 5 m so that aL = 250. A value of ϕ may be obtained from eq. (38) using these values. However, if a = 100 and L = 2.5 so that aL is still equal to 250, an entirely different value of ϕ is obtained because a = 100 rather than 50. One should resist the temptation to multiply numerator and denominator of eq. (38) by L thereby creating a situation where only Z, W and N_c appear along with a new R' = RL. Such a procedure is doomed to failure because in dealing with an equation derived from n equations in n+1 unknowns, one cannot create the n+1th equation by multiplying one of the n equations by a constant. This makes the n+1th equation so obtained linearly dependent on one of the original n equations and the entire set becomes linearly dependent.

1 1 1

This section represents an attempt to obtain $\epsilon = f(R, N_{tu})$ and the attempt has not been successful. It is now time to turn to the computer and this will be done in Section IV.

IV. NUMERICAL AND COMPUTER ANALYSIS

With a closed form solution for ϵ as a function of R and N_{tu} not attainable as indicated in Section III, it becomes apparent that an alternative method is needed to determine the effectiveness for the 1-3:2C and 1-3:2P heat exchangers. Kern and Kraus [Ref. 12: pp. 306-360] describe a computer code for a thermal analyzer. This code (program) makes use of node equations generated by finite differences and it employs a Cholesky LU decomposition scheme.

The Cholesky decomposition, as explained by Stewart [Ref. 13: pp. 134-144], is best used when decomposition in the presence of positive definite matrices is requested. This is the case at hand where one tries to solve numerically for the temperatures that lead to the effectiveness of the 1-3:2C and 1-3:2P heat exchangers.

A. THERMAL ANALYZER TVSSI

The computer program employed is Program TVSSI (Appendix A) which is an adaptation of the thermal analyzer program called TVSS2 and listed by Kern and Kraus [Ref. 12: pp. 306-360]. The adaptation consisted of changing the program so that it could perform the computations in the SI system of units and be receptive to the use of a specially created input file. Also it should be noted that TVSS2 was written

to be used in conjunction with the Honeywell H-1800 computer system. Therefore, it had to be modified to run on the Naval Postgraduate School's IBM 3033 AP system.

The program itself is a non-linear equation solver that determines the temperatures at a prescribed number of nodepoints or nodes from a set of node equations in almost any framework (i.e., network analysis, field plotting, or fluid flow distribution). It has certain features that make it primarily an equation solver for thermal analysis. These features include:

1. an ability to linearize radiation terms.

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- 2. an ability to allow any of the coefficients in the node equations to vary with temperature.
- 3. an ability to provide constant heat input and heat input as a function of temperature at any node.
- 4. an ability to consider other modes of heat transfer that are non-linear such as boiling and natural convection.

As stated earlier, the program utilizes the Cholesky decomposition scheme and, because of the linearization of the radiation terms (a feature of the program that is used even though radiation does not appear in this $\varepsilon - N_{tu}$ study), the program is iterative.

Cholesky's decomposition consists of finding a lower triangular matrix [L] which is capable of reducing the original system of equations.

$$[K][T] = [Q] \tag{40}$$

or

$$[K][T] - [Q] = [0]$$
(41)

to the unit triangular form

$$[A][T] - [B] = [0]$$
(42)

so that the sought after elements of the column vector [T] can be obtained by backward substitution.

Suppose, for example, that [K] is 3x3 and assume that the system [K][T] = [Q] has been reduced to the form [A][T] - [B] = [O]. In this event a premultiplication by [L] will return the system to its original form, that is

$$[L]([A][T] - [B]) = [K][T] - [Q] = [O]$$

This implies that

$$[L][A] = [K]$$
 (43)

and

$$[\mathbf{L}][\mathbf{R}] = [\mathbf{0}] \tag{44}$$

These equations allow the determination of [L], [A], and [B] in a very simple manner and the matrices are uniquely determined because [K] and [Q] are known, or, at least are known after each iteration because the elements of [K] are linearized. For a 3x3 system

[K,Q]					[L]				[A,B]			
k11	k12	k13	91		111	0	0]	1	a12	a13	b1	
k21	^k 22	^k 23	q 2	=	¹ 21	1 ₂₂	0	0	1	a23	^b 2	
k31	k32	k33	q 3		¹ 31	1 ₃₂	1 ₃₃	0	0	1	b3	

one may obtain the following for the elements of [L], [A], and [B].

· · ·

$$k_{11} = (1)l_{11} + (0)l_{12} + (0)l_{13} = l_{11}$$
(45)

which shows that the first column of [L] is identical to the first column of [K].

 $k_{1j} = (l_{11})a_{1j} + (0)a_{2j} + (0)a_{3j} = l_{11}a_{1j} = k_{11}a_{1j}$ (46) which shows that the first row of [A] is equal to the first row of [K] divided by [k_{11}] and then

 $k_{22} = l_{21}a_{12} + l_{22}(1) , \quad l_{22} = k_{22} - l_{21}a_{12}$ $k_{23} = l_{21}a_{13} + l_{22}a_{23} , \quad a_{23} = (k_{23} - l_{21}a_{13})/l_{22}$ $k_{32} = l_{31}a_{12} + l_{32}(1) , \quad l_{32} = k_{32} - l_{31}a_{12}$ $q_{2} = l_{21}b_{1} + l_{22}b_{2} , \quad b_{2} = (q_{2} - l_{21}b_{1})/l_{22}$

In the foregoing manner the elements of [L], [A] and [B] are obtained successively in terms of previously determined elements in a progression that goes horizontally from l_{22} on. Thus the general relationship are seen to be

$$l_{ij} = k_{ij} - \sum_{r=1}^{j-1} l_{ir} k_{rj}$$
(47)

with

$$l_{i1} = k_{i1} \tag{48}$$

and

$$a_{ij} = \frac{1}{I_{ii}} [k_{ij} - \sum_{r=1}^{i-1} l_{ir}a_{rj}]$$
(49)

with

$$\mathbf{a}_{ij} = \frac{\mathbf{k}_{ij}}{\mathbf{k}_{11}} \tag{50}$$

Moreover, it is observed that if [K] is symmetrical which it must be in our coupled set of equations $(k_{i,j} = k_{j,i})$ then

$$l_{ij} = a_{ji}l_{ij}$$
 (i, j = 1, 2, 3, ... n-1; i ≠ j) (51)

The modification for computation in the SI system was quick and simple. It involved changing a numeral in two places (460 to 273) and some format statements (°F to °C and Btu/hr to Watts).

The conversion of TVSS2 to TVSSI for running on the IBM 3033 AP system (which uses the FORTVS compiler) required a modification to the Fortran program language used in TVSS2 in order to compile the FORTVS (basically international Fortran 77) system used on the IBM 3033.

B. INITIAL MODELING

Initially, the model was designed to find the temperatures that would allow computation of a single effectiveness value after a detailed set of capacity rate, coefficients and surface data were entered into the program. From the scope of the problem it was realized that multiple runs of the thermal analyzer program (TVSSI) would be needed. It therefore became necessary to develop a program that given C_h , C_c , U, A, T₁, and t₁, an input file would be created for use by the modified version of the thermal analyzer (TVSSI).

The first step taken was to develop a program to create an input file for TVSSI that would yield the effectiveness for a 1-4 heat exchanger which could be compared to the existing analytical solution for the effectiveness of a 1-4 exchanger. With this accomplished and confidence established, similar programs for the 1-3:2C and 1-3:2P exchangers could be developed. This program was called NTU14 (See Appendix B) and the following parameters were used for all runs.

- 1. 250 nodes were used.
- 2. The initial temperature for the computer to begin the iterative process was set at 200°C.

- 3. An eventual accuracy of .05 between the final and next to last iterations was used.
- 4. A radiation coefficient convergence factor of 0.66667 between iterations was used.
- 5. The maximum number of iterations that the computer was allowed to perform was set at 12.
- 6. A damping factor of .8 was set as an initial damping based on the number of non-linear terms in all of the node equations.

When the values of C_h , C_c , U, A, T_1 and t_1 are set, N_{tu} and R are then compiled and an input file for TVSSI was generated. In this file all node equations and internode conductance values were determined. This program determined and specified the nodes that interact with each other and the methods by which the interaction takes place such as conduction, forced convection, and fluid flow.

The program, NTU14, makes use of the fact that each term in a node equation shows three things. The first is the node that is coupled for heat flow with the node in question. The second is the method of heat flow between the nodes. In this case forced convection and fluid flow are used. Finally, the node equation shows the magnitude of the internode heat flow. Here all the pieces of information are collected and presented for use by TVSSI as an input file with all items in the proper format.

A comparison of the effectiveness for the 1-4 exchanger developed by the computer to that of using the closed form analytical solution for effectiveness developed by Kraus

and Kern [Ref. 8] as shown by equation (52)

$$\varepsilon = \frac{2}{1 + R + \frac{1}{2} [1 + 4R^2]^{1/2} \coth(\frac{N \tan[1 + 4R^2]}{4}) + 4 \tanh \frac{N \tan 1}{4}}$$
(52)

was then undertaken. The results of this comparison showed that over the entire range of R from .01 to 1.0 for varying values of N_{tu} from 0 to 3.25 less than a 0.5% difference was ever realized. A small sample of these results are provided in Table 1. The conclusion to be drawn here, is that the methodology used to develop the computer program NTU14 for input to TVSSI for finding effectiveness was sound and could then be used in the development of the 1-3:2C and 1-3:2P exchanger methodology.

C. DEVELOPED MODELS FOR 1-3:2C AND 1-3:2P HEAT EXCHANGERS

The same technique used in developing the program NTU14 was used to generate computer programs NTU32C and NTU32P. These are listed in Appendices C and D. The departure for each of these programs from the NTU14 program is in the number of nodes; they are based on 200 node models as shown in Figures 4.1 and 4.2. An example of the output file generated from one of these programs is found at Appendix E. It is these values shown in Appendix E that are used by the thermal analyzer to determine the temperatures T₂ and t₂ for the specific set of given initial parameters C_h, C_c, R, A, T₁, and t₁.

1	N	ANALYTICAL	COMPUTER	%		
	"tu	RESULTS	RESULTS	DIFFERENCE		
1						
	0.05	.0482	.0482	0.00		
	0.25	.2094	. 2090	0.20		
	0.50	.3569	. 3559	0.28		
	0.75	.4628	.4612	0.35		
	1.00	. 5398	. 5377	0.39		
	1.25	.5963	. 5940	0.39		
	1.50	.6379	.6354	0.40		
	2.00	.6915	.6886	0.42		
	2.50	. 7206	.7177	0.40		
	3.00	.7360	.7333	0.37		
	3.25	.7406	.7381	0.34		

TABLE 1 COMPUTER TO ANALYTICAL COMPARISON

FOR R = .5

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D. SCOPE OF COMPUTER ANALYSIS

At this point it is possible to let the computer solve for temperature values that yield a value for effectiveness based on a specific set of initial parameters. However, it must be realized that many computer runs are required to generate enough data to ensure confidence in the results which cover a wide range of capacity rate ratios and N_{tu} values.

To efficiently expedite the computer task, the Multiple Virtual System (MVS) with Job Entry Subsystem and Networking (JES3) was utilized. The MVS coupled with JES3 is more commonly referred to as batch processing. Based on trial and error, it was determined that in order to build a solid data base, eleven different values for effectiveness were needed to best represent a particular value of R. This was required over a range of R from R = 0.1 to 1.0 in increments of .01. In all, 200 curves for both the 1-3:2C and 1-3:2P exchangers were needed. This means that 2,200 unique values of effectiveness to be used for comparison with the 1-4 exchangers.

To complete this task, TVSSI was slightly modified in accordance with the appropriate guidelines of the job control language (JCL) needed to run on the batch processing system. These modifications are few and were needed only at the beginning and end of TVSSI. The modified version

of TVSSI has been called TVCOUNT with changes shown in Appendix F. It was TVCOUNT that was then used to activate TVSSI.

It also became necessary to modify the three input file programs NTU14, NTU32P and NTU32C such that they needed to be compiled only once. They were then loaded in a library file to be used when called by another program. New programs utilizing the batch system were written that could easily be loaded with the appropriate input data for a specific R value. These, which are referred to as "sister programs," were used to go from the library file to TVSSI and cause TVSSI to be executed eleven times under TVCOUNT covering the desired range of N_{tu} for a specific R value. The revised input files called NTU14C, NTU32CC, NTU32PC and their associated "sister execution programs," NTU14L, NTU32CL, NTU32PL, are found in Appendices G through L.

The overall system flow chart of how all of the foregoing is accomplished is found in Figures 4.3, 4.4 and 4.5. It is noted from these figures that TVSSI is referred to as TVSSIA through TVSSIV. These are the same programs as TVSSI but for bookkeeping purposes by the computer they are labeled A through V.

E. COMPUTER RESULTS

Upon completion of all data collection from the computer output, plots of effectiveness vs. N_{tu} for the whole range

of R were plotted and are shown in Appendices M and N. Thirty-three different plotting programs utilizing the "Display Integrated Software System and Plotting Language (DISSPLA)," were written to graph the data obtained. An example of one of these programs is provided in Appendix O.

Examination of the graphs in Figures 4.6, 4.7, and 4.8 shows that the 1-3:2C exchangers outperform the 1-3:2P, 1-2 and 1-4 exchangers. This is true in all cases, and, because of this, only a sample of the data was chosen to be shown in these figures. Furthermore, it is noted that at higher N_{tm} values, the effectiveness of the 1-3:2C exchanger is better than all of the others considered, while at higher capacity rate ratios, the effectiveness of 1-3:2C exchanger even begins to outperform the others at lower N_{tu} values. This increase in performance is easily understood because it has been proven by Kern [Ref. 1: pp. 139-137] and others that greater temperature differences result when process streams are in counterflow than parallel flow. Thus, when there is a combination of the two phenomena (counterflow and parallel flow) occurring, then it becomes apparent that the extra counterflow pass must increase the exchanger's overall performance.

As shown in the graphs in Appendices M and N, effectiveness increases as both R and N_{tu} increase. These curves can be used to give a graphical approximation of effectiveness

if that is all that is required. From the empirical data used to develop these curves, further investigation into the development of equations for these curves which may be used to determine an exact value of effectiveness when R and N_{tu} are known, is undertaken in Section V.

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Figure 4.6 Comparison of Analytical (1-2 and 1-4) to Computer Results (1-3:2C and 1-3:2P) at R = .2



Figure 4.7 Comparison of Analytical (1-2 and 1-4) to Computer Results (1-3:2C and 1-3:2P) at R = .5



Figure 4.8 Comparison of Analytical (1-2 and 1-4) to Computer Results (1-3:2C and 1-3:2P) at R = 1.0

V. POLYNOMIAL REGRESSION

A. DEVELOPMENT OF POLYNOMIAL EQUATIONS

The empirical data obtained for the two 1-3 heat exchangers was designed to cover an extensive range of R values varying from 0.01 to 1.0 in increments of 0.01. As discussed earlier in Section IV, the data obtained at a specific value of R is the computer evaluated result of the effectiveness, for an associated N_{tu} value. With this accomplished, it then becomes possible to graph separate curves for each of the different R values as shown in Appendices M and N. Through a polynomial regression technique, as discussed in this section, it is also possible to develop implicit equations for the curves with $\varepsilon = f(N_{tu})$, R). It is also apparent from an inspection of the graphical representation of the empirical data in Appendices M and N, that the curves conform to a high degree polynomial. However, further analytical investigation is needed to ascertain the exact order of the polynomial terms. This investigation will not only lead to the order of the polynomial, but to the specific equation for each curve.

By use of polynomial regression, the least-squares method can be readily extended to best fit the data to the m^{th} -degree for the polynomial

$$y = A_0 + A_1 x + A_2 x^2 + \dots A_m x^m$$
 (53)
with the error defined by

$$e_i = D_i - y_i = D_i - A_0 - A_1 x - A_2 x_i^2 - \dots - A_m x_i^m$$

where D_i represents the empirical data value corresponding to x_i , x_i being free of error.

The objective is to minimize the sum of the squares of the residuals, S_r ,

$$S_{r} = \sum_{i=1}^{m} e_{i}^{2} = \sum_{i=1}^{m} (D_{i} - A_{0} - A_{1}x_{1} - A_{2}x_{2}^{2} + \dots + A_{m}x^{m})^{2}(54)$$

Because at a minimum, the partial derivatives $\partial S_r / \partial A_0$, $\partial S_r / \partial A_1 \dots \partial S_r / \partial A_m$ vanish, after taking the derivative of S_r with respect to each of the coefficients of the polynomial, it can be seen that

$$\frac{\partial S_r}{\partial A_0} = 0 = -2 \Sigma \left(D_i - A_0 - A_1 x_1 - A_2 x_1^2 - \dots - A_m x_i^m \right)$$

$$\frac{\partial S_{r}}{\partial A_{1}} = 0 = -2 \Sigma x_{1}(D_{1} - A_{0} - A_{1}x_{1} - A_{2}x_{1}^{2} - \dots - A_{m}x_{m}^{m})$$

$$\frac{\partial S_{r}}{\partial A_{2}} = 0 = -2 \Sigma x_{i}^{2} (D_{i} - A_{0} - A_{1}x_{i} - A_{2}x_{i}^{2} - \dots - A_{m}x_{i}^{m})$$

$$\vdots$$

$$\frac{\partial S_{\mathbf{r}}}{\partial A_{\mathbf{m}}} = 0 = -2 \Sigma \mathbf{x}_{\mathbf{i}}^{\mathbf{m}} (D_{\mathbf{i}} - A_{\mathbf{0}} - A_{\mathbf{1}}\mathbf{x}_{\mathbf{i}} - A_{\mathbf{2}}\mathbf{x}_{\mathbf{i}}^{\mathbf{2}} - \dots - A_{\mathbf{m}}\mathbf{x}_{\mathbf{i}}^{\mathbf{m}})$$

Then by dividing by -2 and rearranging we obtain $A_0M + A_1 \Sigma x_1 + A_1x_1^2 + \ldots + A_m \Sigma x_1^m = \Sigma D_1$ $A_0 \Sigma x_1 + A_1 \Sigma x_1^2 + A_2 \Sigma x_1^3 + \ldots + A_m \Sigma x_1^{m+1} = \Sigma x_1 D_1$ $A_0 \Sigma x_1^2 + A_1 \Sigma x_1^3 + A_2 \Sigma x_1^4 + \ldots + A_m \Sigma x_1^{m+2} = \Sigma x_1^2 D_1$ \ldots

 $A_0 \Sigma x_i^m + A_1 \Sigma x_i^{m+1} + A_2 \Sigma x_i^{m+2} + \ldots + A_m \Sigma x_i^{2m} = \Sigma x_i^{mD_i}$

where all summations are from i=1 through n. All of the foregoing m+1 equations are linear and have m+1 unknowns: A₀, A₁, A₂, ... A_m. The coefficients of the unknowns can be calculated directly from the observed data. Thus, the problem of determining a least-squares polynomial of degree m is equivalent to solving a system of m+1 simultaneous linear equations. Putting the equations in matrix form yields

From this point on, one finds that it is best to use a computer to assist in solving the simultaneous equations and this will also help alleviate any ill-conditioning that may otherwise occur. An existing "curvefit" program available through NON-IMSL [Ref. 16] and found in Appendix P was used although some modifications were made to the original program to best accommodate the goals of this work.

To determine the order of polynomial that should eventually be used, one increases the degree of the approximating polynomial as long as there is a statistically significant decrease in the variance σ^2 , which is computed by

$$\sigma^2 = \frac{\Sigma e_i}{N - m - 1}$$
(55)

In otherwords, the selection of the optimum degree polynomial is contingent upon a decreasing variance and once the variance begins to increase, the degree of the polynomial becomes too high. For all cases, it was found that the ε - N_{tu} developed curves are of the 5th order.

As shown in Figures 5.1 and 5.2 the computed values of effectiveness vs. N_{tu} for R = 0.1, 0.5 and 1.0 for both flow arrangements, (1-3:2P) and (1-3:2C), have been graphed and fitted by a 5th degree polynomial. Because all computed values for effectiveness follow a predictable trend, only a sample of the data covering the whole range of

values of R have been shown. It is clear that the graphic interpretation strongly backs what is known analytically from the polynomial regression technique. Where the relationship for $\varepsilon = f(N_{tu}, R)$ is found explicitly from

$$\epsilon = A_5 N_{tu}^5 + A_4 N_{tu}^4 + A_3 N_{tu}^3 + A_2 N_{tu}^2 + A_1 N_{tu} + A_0 \quad (56)$$

while the corresponding coefficients A_5 , A_4 , A_3 , A_2 , A_1 , and A_0 relating to a specific value of R are found in Tables 2 and 3 for the (1-3:2P) and (1-3:2C) configurations. An example of how to use this equation in a heat exchanger problem now follows.

B. NUMERICAL EXAMPLE

Consider a heat exchanger containing 400 m² of (A = 400 m^2) of surface and operating with an overall heat transfer coefficient of 80 W/m² °C. Cold fluid at a capacity rate of 10,000 W/°C enters the exchanger at 60 °C. Hot fluid at a capacity rate of 20,000 W/°C enters the exchanger at 200 °C.

1. Find

The effectiveness (ε) and compute the hot and cold fluid outlet temperatures for the (1-3:2C) shell tube pass arrangement.

2. Assumptions

1) Negligible heat loss to surroundings and kinetic and potential energy changes.

- 2) Constant thermal and fluid properties for both fluids.
 - 3. Analysis

Here $C_c = 10,000 \text{ W/}^{\circ}C$ and $C_h = 20,000 \text{ W/}^{\circ}C$ this

makes $R = C_{min}/C_{max} = C_c/C_h = (T_1 - T_2)/(t_2 - t_1)$

10,000/20,000 = 0.5.

and $N_{tu} = UA/C_c$

= 80 (400)/(10,000) = 3.2

First, go to Table 2 (page 81) for the (1-3:2C) arrangement with R = 0.5 and find the coefficients

 $A_0 = 0.1294 \times 10^{-2}$ $A_1 = 0.98120$ $A_2 = -0.66161$ $A_3 = 0.27938$ $A_4 = -0.66456 \times 10^{-1}$ $A_5 = 0.66069 \times 10^{-2}$

Then apply equation (56) for $N_{tu} = 3.2$

 $\varepsilon = A_5 N_{tu}^5 + A_4 N_{tu}^4 + A_3 N_{tu}^3 + A_2 N_{tu}^2 + A_1 N_{tu} + A_0 \quad (56)$ $\varepsilon = 0.769$ Because $C_c < C_h$

$$\varepsilon = \frac{t_2 - t_1}{T_1 - t_1}$$

and with $T_1 - t_1 = 200 - 60 = 140$ °C

$$t_2 - t_1 = \epsilon(T_1 - t_1)$$

= 0.769 (140)
= 107.7°C

Finally, the outlet cold fluid temperature t2 is

$$t_2 = 107.7 + t_1$$

= 107.7 + 60
= 167.7°C

and the fluid temperature, T_2 is easily found

$$R = \frac{T_1 - T_2}{T_2 - T_1} = 0.5$$

$$T_2 = T_1 - 0.5 (t_2 - t_1)$$

= 200 - 0.5 (t_2 - t_1)
= 146 180

4. Observations

The primary observation made here is that by using the 5th order polynomial equation (56) with the appropriate coefficients found in Table 2 or 3, an accurate value for effectiveness can be found thus allowing one to solve for many more unknown values or prarameters of the heat exchanger (i.e., hot and cold outlet temperatures).

The other observation that is to be made is that when comparing the value for effectiveness computed here against the value for a 1-2 or 1-4 exchanger (0.745 and 0.740 respectively) under the same conditions, one finds that there is a significant difference in exchanger performance as a function of odd or even tube passes and that the 1-3:2C arrangement has a higher effectiveness than either the 1-2 or 1-4 arrangement. From inspection of the curves for the 1-3:2P exchanger at Figure N-1 or N-6 with R = 0.5 and N_{tu} = 3.2 an approximate value of ε = .715 is obtained. It is clear that this is also less than that of 1-3:2C arrangement. Therefore, it is evident that the 1-3:2C exchanger out-performs not only the 1-2 and 1-4 arrangement but also its counterpart the 1-3:2P exchanger by 3.1%, 3.8% and 7.0% respectfully.



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Figure 5.1 1-3:2C Data Fit by a 5th Order Polynomial





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TABLE 2 1-3:2C COEFFICIENTS

		_				_	_					_	~	_		_			_	-
<u> 45 x10²</u>	0.27562	0.19915	0.23943	0.25908	0.24608	0.25657	0.28161	0.26475	0.29217	0.28264	0.30429	0.31765	0.32373	0.31186	0.31623	0.33043	0.32587	0.34412	0.22473	
A4 x10 ¹	-0.31787	-0.25435	-0.28857	-0.30646	-0.29488	-0.30594	-0.32725	-0.31420	-0.33695	-0.33137	-0.34744	-0.35920	-0.36561	-0.35696	-0.36288	-0.37606	-0.37211	-0.38883	-0.30036	
<u>8</u>	0.16418	0.14498	0.15565	0.16165	0.15792	0.16207	0.16867	0.16524	0.17208	0.17131	0.17555	0.17938	0.18182	0.17971	0.18229	0.18673	0.18557	0.19109	0.16874	
<u>-</u> 2	-0.51115	-0.48530	-0.49950	-0.50794	-0.50226	-0.50845	-0.51698	-0.51309	-0.52158	-0.52163	-0.52589	-0.53101	-0.53462	-0.53226	-0.53636	-0.54245	-0.54081	-0.54820	-0.52634	
t\$	1.00630	0.98977	0.99515	0.99772	0.99188	0.99329	0.99546	0.99167	0.99354	0.99198	0.99116	0.99173	0.99173	0.98852	0.98885	0.98999	0.98713	0.98875	0.98061	
	-1.54650	1.09620	0.23928	-0.23948	0.70811	0.36469	0.17636	0.62843	0.41140	0.47085	0.68979	0.53544	0.38892	0.76975	0.60634	0.40649	0.73099	0.46417	0.90075	
	.01	.02	.03	.04	.05	.06	.07	.08	.09	.10	.11	.12	.13	.14	.15	.16	.17	.18	.19	_

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A 5 x10 ²	0.35808	0.37222	0.38422	0.38931	0.40315	0.40637	0.41785	0.43663	0.44549	0.46888	0.51644	0.46289	0.46752	0.48114	0.49414	0.49105	0.49765	0.50506	0.50798	0.51644
$\mathbf{A_4 \times 10^1}$	-0.40158	-0.41317	-0.42439	-0.42850	-0.44087	-0.44493	-0.45381	-0.46901	-0.47655	-0.49504	-0.52608	-0.49281	-0.49644	-0.50820	-0.51887	-0.51751	-0.52330	-0.53023	-0.53284	-0.54058
A3	0.19530	0.19879	0.20258	0.20381	0.20786	0.20960	0.21210	0.21654	0.21893	0.22410	0.23115	0.22464	0.22572	0.22942	0.23262	0.23267	0.23460	0.23702	0.23792	0.24047
A2	-0.55346	-0.5575	-0.56291	-0.56423	-0.56969	-0.57241	-0.57511	-0.58041	-0.58350	-0.58911	-0.59584	-0.59141	-0.59256	-0.59730	-0.60118	-0.60162	-0.60424	-0.60770	-0.60875	-0.61207
	0.98703	0.98700	0.98762	0.98611	0.98692	0.98653	0.98547	0.98591	0.98553	0.98544	0.98625	0.98380	0.98226	0.98264	0.98243	0.98083	0.98036	0.98036	0.97884	0.97855
$A_0 \times 10^3$	0.71140	0.64893	0.56702	0.71515	0.61725	0.57119	0.79180	0.69607	0.62722	0.79437	0.67158	0.74865	0.92219	0.82224	0.88180	0.10262	0.99038	0.91970	1.08830	1.11470
8	.20	.21	.22	.23	.24	.25	.26	.27	.28	.29	.30	.31	.32	.33	.34	.35	.36	.37	.38	.39

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10 ¹ A ₅ x10 ²	30 0.51566	28 0.52441	18 0.53262	90 0.54017	28 0.53543	29 0.53694	73 0.55343	04 0.55347	81 0.55130	76 0.90142	56 0.66069	97 0.56948	67 0.58553	18 0.58416	66 0.59853	51 0.61405	82 0.60403	07 0.61674	
¥-4	-0.541	-0.548	-0.556	-0.562	-0.560	-0.563	-0.576	-0.577	-0.576	-0.891	-0.664	-0.593	-0.606	-0.607	-0.618	-0.631	-0.624	-0.635	
	0.24125	0.24330	0.24606	0.24826	0.24798	0.24946	0.25345	0.25368	0.25421	0.35867	0.27938	0.26012	0.26376	0.26447	0.26778	0.27164	0.27025	0.27325	
A2	-0.61372	-0.61606	-0.61993	-0.62278	-0.62296	-0.62533	-0.63022	-0.63045	-0.63176	-0.77159	-0.66161	-0.63972	-0.64398	-0.64538	-0.64922	-0.65396	-0.65281	-0.65632	
	0.97771	0.57688	0.97705	0.97657	0.97508	0.97449	0.97496	0.97310	0.97221	0.94641	0.98102	0.97076	0.97079	0.96986	0.96971	0.97013	0.96803	0.96775	
A0 ×10 ³	1.17740	1.21820	1.13560	1.17390	1.25770	1.30410	1.20250	1.39770	1.45510	2.63160	1.29400	1.46920	1.50140	1.54710	1.56460	1.46430	1.65860	1.68550	
	.40	.41	.42	.43	.44	.45	.46	.47	.48	.49	.50	.51	.52	.53	.54	.55	.56	.57	1

A5 x10 ²	0.62630	0.61866	0.63700	0.64910	0.65496	0.66649	0.67144	0.67199	0.68424	0.69110	0.70539	0.70618	0.70940	0.71533	0.72461	0.72971	0.73945	0.74106	0.74373	0.76409
A4 x10 ¹	-0.64435	-0.64120	-0.65464	-0.66476	-0.66975	-0.67963	-0.68455	-0.68584	-0.69586	-0.70174	-0.71344	-0.71473	-0.71834	-0.72405	-0.73180	-0.73620	-0.74462	-0.74680	-0.74972	-0.76577
A3.	0.27651	0.27649	0.28014	0.28319	0.28478	0.28786	0.28960	0.29033	0.29330	0.29518	0.29860	0.29922	0.30065	0.30265	0.30501	0.30640	0.30908	0.31004	0.31115	0.31567
A2	-0.66068	-0.66124	-0.66567	-0.66936	-0.67135	-0.67523	-0.67750	-0.67878	-0.68231	-0.68466	-0.68863	-0.68955	-0.69161	-0.69436	-0.69723	-0.69884	-0.70230	-0.70372	-0.70520	-0.71036
A1	0.96636	0.96466	0.96543	0.96535	0.96455	0.96471	0.96393	0.96311	0.96293	0.96231	0.96227	0.96109	0.96047	0.96018	0.95987	0.95882	0.95886	0.95796	0.95700	0.95754
A ₀ ×10 ³	1.77010	1.85570	1.76480	1.74620	1:80880	1.72290	1.85600	1.81240	1.82510	1.87880	1.88370	1.96850	1.98700	2.00210	1.95690	2.08210	2.00140	2.07320	2.14030	2.10090
R	.59	.60	.61	.62	.63	.64	.65	.66	.67	.68	.69	.70	.71	.72	.73	.74	.75	.76	.77	.78

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	-0.76691 0.76409	-0.77811 0.77833	-0.77439 0.77261	-0.78513 0.78474	-0.78494 0.78416	-0.79853 0.80160	-0.80318 0.80576	-0.80894 0.81289	-0.81036 0.81397	-0.81554 0.81943	-0.81640 0.81960	-0.82268 0.82712	-0.83276 0.83937	-0.83576 0.84233	-0.84485 0.85293	-0.84835 0.85688		-0.85120 0.85945	-0.85120 0.85945 -0.85586 0.86503
	0.31638	0.31955	0.31881	0.32224	0.32235	0.32610	0.32788	0.32960	0.33023	0.33197	0.33251	0.33440	0.33739	0.33847	0.34133	0.34240	0 36350	300000	0.34495
A2	-0.71149	-0.71500	-0.71442	-0.71882	-0.71902	-0.72313	-0.72564	-0.72764	-0.72849	-0.73069	-0.73156	-0.73371	-0.73726	-0.73861	-0.74226	-0.74336	-0 74494		-0.74656
	0.95647	0.95612	0.95454	0.95505	0.95352	0.95366	0.95324	0.95252	0.95135	0.95078	0.94972	0.94902	0.94906	0.94803	0.94826	0.94709	0.94638		0.94547
	2.17470	2.22410	2.29050	2.17700	2.36540	2.25330	2.31490	2.33970	2.42210	2.46190	2.50950	2.55370	2.48350	2.62780	2.53660	2.62050	2.63490		2.70740
×; ·	.79	.80	.81	.82	.83	.84	.85	.86	.87	.88	.89	.90	16.	.92	.93	.94	.95		.96

TABLE 2 (cont'd)

1-3:2C COEFFICIENTS

A5_x10 ²	0.88147	0.89055	0.89109
A4_x101	-0.86995	-0.87737	-0.87876
A3	0.34931	0.35148	0.35222
_ A2	-0.75178	-0.75421	-0.75536
	0.94473	0.94417	0.94336
Å0_x10 ³	2.75280	2.79380	2.77220
8	.98	66.	1.0

TABLE 3 1-3:2P COEFFICIENTS

— I	-							_												_	
<u> A5_x10</u> 2	_	0.27066	0.19708	0.23299	0.24901	0.23286	0.28450	0.28805	0.27915	0.27957	0.28716	0.30175	0.32827	0.30842	0.29604	0.31312	0.32851	0.33344	0.35028	0.35493	
A4 x10 ¹		-0.31456	-0.25245	-0.28333	-0.29827	-0.28564	-0.32915	-0.33265	-0.32501	-0.32731	-0.33459	-0.34560	-0.36872	-0.35466	-0.34561	-0.36119	-0.37453	-0.37882	-0.39372	-0.39841	
A3		0.16346	0.14443	0.15419	0.15936	0.15581	0.16930	0.17043	0.16820	0.16963	0.17231	0.17532	0.18264	0.17935	0.17711	0.18226	0.18651	0.18791	0.19272	0.19446	
I	_	-0.51077	-0.48504	-0.49839	-0.50600	-0.50132	-0.51965	-0.52063	-0.51784	-0.52061	-0.52472	-0.52796	-0.53763	-0.53466	-0.53222	-0.53929	-0.54496	-0.54674	-0.55323	-0.55579	
A1	_	1.00620	0.98955	0.99459	0.99665	0.99151	0.99957	0.99690	0.99321	0.99282	0.99298	0.99174	0.99423	0.99141	0.98808	0.98956	0.99034	0.98887	0.99011	0.98934	
A0_x10 ³		-1.53750	1.11580	0.26058	-0.14339	0.71311	-0.76150	0.00791	0.53649	0.45711	0.32341	0.63371	0.44811	0.40379	0.80291	0.55870	0.39008	0.61833	0.37408	0.44322	
- R	-	.01	.02	.03	.04	.05	.06	.07	.08	60.	.10	.11	.12	.13	.14	.15	.16	.17	.18	.19	

2	A0 x10 ³	A1	A2	A3.	A4 x10 ¹	A5 x10 ²
	0.65879	0.98800	-0.55782	0.19608	-0.40326	0.36012
.21	0.62161	0.98815	-0.56240	0.19955	-0.41423	0.37300
.22	0.52051	0.98852	-0.56700	0.20282	-0.42390	0.38353
.23	0.70119	0.98663	-0.56742	0.20307	-0.42417	0.38350
.24	0.55381	0.98769	-0.57404	0.20845	-0.44276	0.40697
.25	0.50155	0.98755	-0.57772	0.21100	-0.44987	0.41422
.26	0.75200	0.98606	-0.57918	0.21217	-0.45355	0.41867
.27	0.67777	0.98627	-0.58389	0.21587	-0.46579	0.43366
.28	0.64963	0.98566	-0.58598	0.21689	-0.46711	0.43334
.29	0.89133	0.98422	-0.58768	0.21821	-0.47081	0.43704
.30	0.83692	0.98406	-0.59124	0.22081	-0.47884	0.44648
.31	0.72206	0.98469	-0.59633	0.22452	-0.49011	0.45909
.32	0.88562	0.98325	-0.59765	0.22550	-0.49292	0.46229
.33	0.81793	0.98323	-0.60125	0.22803	-0.50028	0.47027
.34	0.93507	0.98177	-0.60195	0.22811	-0.49868	0.46646
.35	1.02360	0.98140	-0.60574	0.23118	-0.50876	0.47864
.36	1.00020	0.98080	-0.60808	0.23263	-0.51221	0.48161
.37	0.93995	0.98064	-0.61149	0.23506	-0.51930	0.48934
.38	1.03810	0.98003	-0.61416	0.23693	-0.52453	0.49483
.39	1.12760	0.97855	-0.61487	0.23720	-0.52419	0.49327

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A5 ×10 ²	0.50692	0.50348	0.51805	0.51682	0.53160	0.53512	0.54187	0.55186	0.55729	0.56316	0.61370	0.57445	0.57989	0.59746	0.59779	0.60093	0.61487	0.62372	0.62022	
A4 x10 ¹	-0.53686	-0.53399	-0.54598	-0.54612	-0.55858	-0.56229	-0.56938	-0.57754	-0.58269	-0.58859	-0.63149	-0.59899	-0.60437	-0.61662	-0.61975	-0.62353	-0.63438	-0.64250	-0.64102	
<u>A</u> 3	0.24148	0.24070	0.24433	0.24480	0.24865	0.25010	0.25277	0.25522	0.25705	0.25916	0.27237	0.26273	0.26428	0.26750	0.26958	0.27113	0.27418	0.27690	0.27699	
A2	-0.62069	-0.61982	-0.62437	-0.62536	-0.63029	-0.63244	-0.63644	-0.63939	-0.64198	-0.64490	-0.66207	-0.64974	-0.65249	-0.65450	-0.65881	-0.66116	-0.66454	-0.66822	-0.66891	
t	0.97929	0.97718	0.97741	0.97606	0.97648	0.97568	0.97599	0.97531	0.97482	0.97436	0.98069	0.97298	0.97251	0.97061	0.97180	0.97129	0.97070	0.97074	0.96951	
	1.11050	1.21970	0.13290	1.23020	1.19100	1.25920	1.16020	1.29110	1.22590	1.26780	0.88037	1.35630	1.39390	1.56550	1.45620	1.39610	1.52440	1.45420	1.51230	
	.40	.41	.42	.43	44.	.45	.46	.47	.48	.49	.50	.51	.52	.53	.54	.55	.56	.57	.58	_

TABLE 3 (cont'd)

1-3:2P COEFFICIENTS

— I					_						-						-			
A 5 x10 ²	0.62890	0.69564	0.64118	0.64908	0.65501	0.67424	0.66610	0.67579	0.68609	0.69188	0.69995	0.70233	0.70328	0.71559	0.72452	0.73176	0.73913	0.73960	0.75395	0.76062
A4 x101	-0.64826	-0.70007	-0.66030	-0.66713	-0.67259	-0.68781	-0.68275	-0.69143	-0.70005	-0.70500	-0.71240	-0.71503	-0.71702	-0.72691	-0.73490	-0.74079	-0.74750	-0.74887	-0.76034	-0.76647
A3	0.27925	0.29329	0.28351	0.28565	0.28752	0.29187	0.29094	0.29381	0.29643	0.29800	0.30047	0.30149	0.30255	0.30541	0.30803	0.30976	0.31201	0.31279	0.31610	0.31810
A 2	-0.67179	-0.68701	-0.67761	-0.68023	-0.68272	-0.68781	-0.68713	-0.69099	-0.69417	-0.69611	-0.69931	-0.70067	-0.70246	-0.70572	-0.70916	-0.71106	-0.71407	-0.71534	-0.71914	-0.72159
A1	0.96909	0.97304	0.96830	0.96764	0.96705	0.96755	0.96555	0.96574	0.96543	0.96457	0.96428	0.96317	0.96250	0.96210	0.96211	0.96106	0.96092	0.95993	0.95981	0.95917
$A_0 \times 10^3$	1.54580	1.29670	1.58550	1.64580	1.69090	1.58780	1.78040	1.69280	1.72500	1.76970	1.78510	1.87300	1.89020	1.92640	1.85070	2.01410	1.91680	1.98610	2.00770	2.05850
R	.59	.60	.61	.62	.63	.64	.65	.66	.67	.68	.69	.70	.71	.72	.73	.74	.75	.76	.77	.78

R $A_0 \times 10^3$ A_1 A_2 A_3 A_{10^4} $A_5 \times 10^4$ $A_5 \times 10^3$.79 2.14320 0.95793 -0.72227 0.31840 0.78210 0.78210 .80 2.14320 0.95746 -0.72853 0.32334 -0.78412 0.78210 .81 2.15380 0.957746 -0.72853 0.32558 -0.78896 0.77874 .82 2.11630 0.95557 -0.73167 0.32558 -0.78896 0.78556 .83 2.27160 0.95557 -0.73167 0.32558 -0.78896 0.78556 .84 2.19230 0.955480 -0.73167 0.332396 -0.78851 0.79585 .85 2.25020 0.95441 -0.74010 0.33290 -0.81186 0.80142 .86 2.256020 0.95441 -0.74139 0.33537 -0.81186 0.81156 .87 2.31220 0.95544 -0.74139 0.33537 -0.82142 0.80142 .88 2.39540 0.955146 0.74728 0.33550 -0.82797 0.82834 .88 2.3			_						_	_		_	_		_	_	_	-		_	_
R A0 x10 ³ A1 A2 A3 A4 x10 ⁴ 79 2.14880 0.95793 -0.72227 0.31840 -0.76612 .81 2.14320 0.95746 -0.72263 0.32352 -0.78896 .81 2.15380 0.95704 -0.73167 0.32358 -0.78896 .82 2.11630 0.95557 -0.73167 0.32558 -0.78896 .83 2.27160 0.95559 -0.73167 0.32558 -0.78896 .83 2.219230 0.95440 -0.73116 0.32598 -0.78958 .84 2.19230 0.95440 -0.7411 0.332875 -0.78958 .84 2.19230 0.95440 -0.74399 0.33269 -0.790345 .86 2.238490 0.95441 -0.74728 0.33269 -0.82797 .88 2.39540 0.95284 -0.74728 0.33662 -0.82797 .88 2.39540 0.95284 -0.74728 0.33662 -0.82797 .89 2.39540 0.95284 -0.74728 0.33662 -0.82797	A5 x10 ²	0.75884	0.78210	0.77874	0.78526	0.78567	0.79585	0.80142	0.81156	0.82103	0.82142	0.83153	0.82834	0.84022	0.84728	0.86088	0.85991	0.86383	0.87578	0.88020	
R A_0 $\times 10^3$ A_1 A_2 A_3 .79 2.14880 0.95793 -0.72227 0.31840 .80 2.14320 0.95793 -0.72257 0.31840 .81 2.15380 0.95746 -0.72853 0.323558 .81 2.15380 0.95746 -0.72163 0.32558 .82 2.11630 0.95557 -0.73167 0.32558 .83 2.25160 0.95557 -0.73167 0.32558 .84 2.19230 0.95559 -0.73167 0.325875 .85 2.25020 0.95544 -0.74010 0.332598 .86 2.19230 0.95544 -0.743167 0.33259 .87 2.33540 0.95544 -0.743167 0.33359 .88 2.33540 0.95544 -0.744307 0.33359 .87 2.31220 0.955293 -0.74728 0.33469 .88 2.35490 0.95284 -0.74729 0.34167 .90 2.39460 0.95284 -0.74725 0.34167 .91 2.39460 0.95284 <th>A4 x10¹</th> <th>-0.76612</th> <th>-0.78412</th> <th>-0.78281</th> <th>-0.78896</th> <th>-0.78968</th> <th>-0.79851</th> <th>-0.80345</th> <th>-0.81186</th> <th>-0.81990</th> <th>-0.82109</th> <th>-0.82970</th> <th>-0.82797</th> <th>-0.83821</th> <th>-0.84404</th> <th>-0.85500</th> <th>-0.85550</th> <th>-0.85936</th> <th>-0.86925</th> <th>-0.87333</th> <th></th>	A4 x10 ¹	-0.76612	-0.78412	-0.78281	-0.78896	-0.78968	-0.79851	-0.80345	-0.81186	-0.81990	-0.82109	-0.82970	-0.82797	-0.83821	-0.84404	-0.85500	-0.85550	-0.85936	-0.86925	-0.87333	
R AQ $x10^3$ A_1 A_2 .79 2.14880 0.95793 -0.72263 .80 2.14320 0.95704 -0.72763 .81 2.14320 0.95704 -0.72853 .81 2.14320 0.95704 -0.73116 .82 2.11630 0.95557 -0.73167 .83 2.27160 0.95559 -0.733167 .84 2.19230 0.95559 -0.733167 .84 2.19230 0.95559 -0.733167 .85 2.21600 0.95559 -0.74010 .85 2.235920 0.95444 -0.74307 .86 2.33540 0.955284 -0.74728 .87 2.31220 0.955284 -0.74728 .88 2.35970 0.95284 -0.74728 .90 2.47650 0.95284 -0.74728 .91 2.3450 0.95131 -0.74728 .91 2.3450 0.95131 -0.74728 .91 2.3450 0.95131 -0.77778 .92 2.53450 0.955072	<u>-</u>	0.31840	0.32334	0.32352	0.32558	0.32598	0.32875	0.33036	0.33290	0.33537	0.33602	0.33869	0.33850	0.34167	0.34342	0.34662	0.34718	0.34854	0.35153	0.35284	
R A0 ×10 ³ A1 .79 2.14880 0.95793 .80 2.14320 0.95746 .81 2.15380 0.95746 .82 2.11630 0.95764 .83 2.11630 0.95557 .84 2.19230 0.95557 .85 2.11630 0.95559 .84 2.19230 0.95557 .85 2.2160 0.95557 .86 2.19230 0.95559 .87 2.19230 0.95559 .88 2.19230 0.95440 .87 2.25920 0.954811 .88 2.39540 0.95284 .88 2.35450 0.95284 .90 2.47650 0.95151 .91 2.39540 0.95284 .91 2.39540 0.95151 .91 2.353450 0.95060 .92 2.53450 0.95072 .93 2.49640 0.95072 .94 2.49640 0.956072 .95 2.55890 0.94967 <t< th=""><th> A2</th><th>-0.72227</th><th>-0.72763</th><th>-0.72853</th><th>-0.73116</th><th>-0.73167</th><th>-0.73515</th><th>-0.73710</th><th>-0.74010</th><th>-0.74307</th><th>-0.74399</th><th>-0.74728</th><th>-0.74725</th><th>-0.75112</th><th>-0.75308</th><th>-0.75683</th><th>-0.75778</th><th>-0.75948</th><th>-0.76309</th><th>-0.76451</th><th></th></t<>	A2	-0.72227	-0.72763	-0.72853	-0.73116	-0.73167	-0.73515	-0.73710	-0.74010	-0.74307	-0.74399	-0.74728	-0.74725	-0.75112	-0.75308	-0.75683	-0.75778	-0.75948	-0.76309	-0.76451	
R A0 ×10 ³ .79 2.14880 .80 2.14320 .81 2.15380 .82 2.11630 .83 2.15380 .83 2.15380 .83 2.11630 .84 2.19230 .85 2.11630 .83 2.25020 .84 2.19230 .85 2.25020 .86 2.25020 .87 2.39540 .88 2.39540 .89 2.35970 .91 2.39540 .91 2.39540 .91 2.39540 .92 2.47650 .93 2.47650 .91 2.35890 .92 2.53450 .93 2.47660 .93 2.47660 .93 2.47660 .93 2.47660 .93 2.53450 .94 2.49640 .95 2.55890 .96 2.54640 .97 2.55890		0.95793	0.95825	0.95746	0.95704	0.95557	0.95559	0.95480	0.95444	0.95411	0.95293	0.95284	0.95131	0.95151	0.95060	0.95072	0.94967	0.94885	0.94891	0.94781	
. 79 . 81 . 82 . 83 . 83 . 83 . 83 . 83 . 83 . 83 . 83		2.14880	2.14320	2.15380	2.11630	2.27160	2.19230	2.25020	2.28490	2.31220	2.39540	2.36970	2.47650	2.39460	2.53450	2.42020	2.49640	2.55890	2.54640	2.63240	
	R	.79	.80	.81	.82	.83	.84	.85	.86	.87	.88	.89	.90	16.	.92	.93	.94	.95	.96	.97	

TABLE 3 (cont'd) 1-3:2P COEFFICIENTS[.]

I — I	
<u> 45_x102</u>	0.88592 0.89273 0.90142
A4 x101	-0.87773 -0.88404 -0.89176
A 3	0.35409
	-0.76580 -0.76845 -0.77159
[1	0.94678 0.94670 0.94641
A0_x103	2.68180 2.69800 2.63160
2	.98 .99 1.0

VI. CONCLUSIONS AND RECOMMENDATIONS

A. CONCLUSIONS

Effectiveness values, though not analytically derived, can be determined for the 1-3:2C and 1-3:2P heat exchangers by utilizing a 5th order polynomial approximation. Sufficient data now exists to cover a complete range of capacity rate ratios for values of N_{tu} from 0.0 to 3.25. From the knowledge of a particular three tube pass heat exchanger arrangement and dimensions, fluid flow rates and temperature extremes, N_{tu} and R values may be computed. Then the effectiveness may be determined using the appropriate coefficients from the tables provided herein. After the effectiveness is obtained, there are two choices.

- 1) From a knowledge of q_{max} (see Section III), it is a simple matter to determine the actual heat transfer rate from the postulation that $q = \alpha_{max}$.
- 2) Both fluid outlet temperatures may be computed since $q = \omega_c C_{ph}(t_2 t_1)$.

It is also apparent from the work done here that the 1-3:2C exchanger effectiveness outperforms that of the 1-2, 1-4 and its counter part, the 1-3:2P exchanger, as N_{tu} increases. This is true for any and all values of R. Therefore, because it is possible to determine the effectiveness of a 1-3 exchanger which has a higher effectiveness (the 1-3:2C arrangement) than that of the 1-2, 1-4 and

1-3:2P exchanger particularly at high N_{tu} levels, the 1-3 exchanger can now be given full consideration in heat exchanger design. This would be especially helpful where three tube passes could alleviate a configuration problem.

B. RECOMMENDATIONS

The following recommendations are provided for possible follow-on projects of a similar nature.

- Continue study of the ϵ N_{tu} method for the 1-5 heat exchangers.
- Investigate the possibility of using a linear approximation from the 1-3 data to find a periodicity of R values at which to develop a 1-5 data base.
- Develop interactive software to be used on a microcomputer that contains both the 1-3 and 1-5 polynomial approximation coefficients. The procedure of entering the values for N_{tu} , R, and the type of heat exchanger when asked to do so in a menu-driven fashion so that effectiveness values can be readily obtained is self-evident.
- After results of 1-5 exchanger analyses are complete, investigate the effectiveness for both the 1-3 and 1-5 heat exchangers experimentally to confirm or refute the theoretical work done here.

APPENDIX A

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THERMAL ANALYZER TVSSI COMPUTER PROGRAM

//OHARE JOB (2323, V26/), TVSSL, CLASS=C // #MAIN ORG=NPGVM1.2323P,SYSTEM=SY2 // EXEC FORTVCG //FORT.SYSPRINT DD DUMMY	<pre>//FORT.SYSIN DD * C<<<<<<<<<<<<<<<<<<<<<<<<<<<<<<<<<<<<</pre>	rkugkan 1vaal C ************************************	C * UPDATED APRIL 1985 C * NOTE: IF DAMPING VALUES AND INITIAL TEMPS ARE THE SAME FOR ALL NODE C * ENTER 1 DAMPING VALUE COLS 25-32 CARD 3 C * ENTER 1 INITIAL TEMP COLS 33-40 CARD 3 C * CALL FOR DATA SET 6 ON CARD 2	0 ************************************	C 2 2 2	C * JCL NOTE: LISTING OF INPUT IS WRITTEN ON UNIT 3 BEFORE NORMAL C * OUTPUT ON UNIT 8. TO DELETE INPUT LIST, INSERT DUMMY C * DATA DEFINITION STATEMENT		<pre>DIMENSION TCO(5), CONTMP(50) TTAG(100), TCOEF(90); COEF(100), 1 H(316) (316), TOLD(315) toLD(315) toLD1(315); BETA(315); 2 TITLE(20) INPTAG(10) EX(4), TMPHT(90), TOLD2(315); NOCONT(6); 3 HTR(42) bTUCRV(5) TMPCRV(5), TIMCO(5), TMPHTV(5); NOCONT(6); CHARACTER *4 HEAD1, BETA 'TOLD'/ 1000 READ(40500, END=999) TITLE 1000 READ(40500, END=999) TITLE</pre>	ŘEWIND 1 REWIND 2 READ(4, 501) N,NCT,NOHTRS,NOEXP,NOCASE,NTCOEF,NODCFH,NTMPHT NP1 = N + 1 READ (4,501) NTAG8,NTAG9,NTAG11
---	--	---	---	--	------------------	--	--	---	---



INI

TITLE NCT, NOHTRS, NOEXP, NOCASE, NTCOEF, NODCFH, NTMPHT, NONODS, NITS HTR, CASBTU, NOCONT, TOLD, NODCFH) BTUCRV (NTTH)*COEF (IWD) ŘĚAĎ (4,502) TIMCO CONTINUE CALL TVPAGE CALL TVSOUT(N, 1,TOLD,TOLD,TITLE) id; (BETA(I), I=1,N) TMP (TOLD(I), I=1, N) GO TO 105 TO 55 60 TO 50 AG8) GO TU R NOD, NODEI, NTAC 10) - NOHTR E.0) GO TO 49 F.5) . GO TO 820 GO TC GO TO 830 - NOCT .0) GO TO 60 TMPCRV BTUCRV COEF INI Q.999 A(NP1) 1502) 15 (8,533) 02) READ GO TO WRITE STOP 809 810 808 25 24 25 811 15 17 815 820

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MICROCOPY RESOLUTION TEST CHART NATIONAL BUREAU OF STANDARDS - 1963 - A







NOEXP NOCASE NTCOEF NODCFH NTMPHT NONO RVES (SET 7)') FIX DECK AND RESUBMIT 15X NUMBER - YOU SPECT TO A NODE - I YOU SPECIFIED / O A NODE 15./15X 1X,F11,4)) 16F8;4)) X,16G8.2)) $(\tilde{I})^{\circ}$, LT, 0001) BETA(I), I = 1, 0001 $\tilde{I} = 1, N$ TIÔNS''// NŎĎE , , NODE 4 6612. 612. 6613 TITLE N TOLD(I),I=1,N) L, NCT) S NO NCT NOHI MNITS // ޱ1 NTMP(I) **FRR** NOHTR ÅRE 500 501 6801 FORMAT FORMA URI COLL 630**_**FQ 640_FQ 175 590 610 20055 2005 2005 620 553 009 0000 00000000000

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NPUT FOR THIS PROBLEM.' ATIONS FOR THIS PROBLEM HERE ARE ONLY /15X. LPHA, BETA, TN, TNMI, TNM2, ERR H 0-2 I(315) . TNM2/315 . -----E CO TO 10 E ERR) GO TO 50 PHA) BETA(I) = -BETA(I)*ALPHA/GAMMA , TNM2(315) , BETA(315) ÑM2(I)) T12 = SIGN(1E-06,T12 TEMP-DEPENDENT ,SAVE(49770) NOY $\begin{array}{c} \text{GO TO 50} \\ \text{ALPHA} \\ \text{DETA(I)} = 1 \end{array}$ GO TO 45 3161) 3161) GO TO 50 (K=1,NP1) (EQ. 0.) 9F8.2) SUBROUTI SUBROI DIMEN LOCS (NMI **RETU** END 680 999 50 670 10 650 660 10

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OHTRS, HTR, CASBTU, NOCONT, TOLD, NODCFH) A-H, 0-Z IOCONT(6), TOLD(315) $(T_{1}^{T})^{TCC}$ + $(T_{1}^{TCC})^{TCC}$ + $(T_{1}^{TCC})^{TCC}$ + $(T_{1}^{TCP})^{TCM}$ + $(T_{1}^{CM})^{TCM}$],T2,TITLE)),ID(12),TITLE(20) 200 (†1(K), K= I,NI) $\left\{ T_{2}^{I}\left\{ K\right\} ;\begin{array}{c} K = I \\ K = I \\ M I \end{array} \right\}, N_{1}^{I} = \left\{ N_{1}^{I}\right\}$ ID(K), K=1,N5) 5,5,10,TITLE) TVPAGE (2, TITLE) **IG TEMPS** rvsou REAL (315) COEF (K+1) CO^T=^TTCOEF(NE+1) RETURN END Ī N, 12 ĀĢĘ TE (8,504) 10,50 10,50 10,503 10,503 10,503 10,503 10,503 10,503 10,503 10,503 10,504 10,504 10,504 10,504 10,504 10,504 10,504 10,504 10,504 10,506 10,5 SUBROUTINE T IMPLICIT DIMENSION CASBTU = SUBROUT FORMAT NO n 20 50 60 25 30 50 210 50001000 5000100 5000100

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APPENDIX B

NTU14 COMPUTER GENERATED INPUT ANALYZER PROGRAM

//OHARE JOB (2323,0267),'NTU14',CLASS=B //*MAIN ORG=NPGVM1.2323P EXEC FORTVCL,PARM.LKED='LIST,MAP' C/FORT.SYSIN DD* C	C IT GENERATES AN INPUT FILE FOR THERMAL ANALYZER TO VERIFY C I-4 EFFECTIVENESS-NTU RELATIONSHIP THAT IS AVAILABLE IN C OPEN LITERATURE.	DIMENSION COEF(250,5),KCON(250,5),L1(8),L2(3),L3(6),SET2(2),FL	C CHARACTER *79 TITLE CHARACTER *12 FNAME	C DATA IOT, IN, IPR, IWR/6,5,4,8/	Ç OPEN PRINTER OUTPUT FILE	C OPEN(IPR,FILE='PRN',STATUS='NEW',FORM='FORMATTED',IOSTAT=ICK) IF(ICK.NE.0) WRITE(10T,920) 920 FORMAT('Trouble opening printer output file') C	917 WRITE(IOT,917) 917 FORMAT(/ Input the title of this study 79 columns only.', & This title, will appear',/, at the top of every printed page READ(IN,918): TITLE	918 FORMÁT(Á79) C	901 FORMAT(/7,901) 901 FORMAT(/7,1NPUT HOT SIDE CAPACITY RATE:') 902 FORMAT(BN,F10.0)	903 FCRMAT(/7,1NPUT COLD SIDE CAPACITY RATE:') READ(IN,902) CCLD	
	FY	,FL4(4)				K)	, påge',				

FORMAT(/'INPUT OVERALL HEAT TRANSFER COEFFICIENT:') READ(IN,902) U WRITE(10T,905) FORMAT(//T.NPUT TOTAL HEAT TRANSFER SURFACE:') READ(1N,902) SURFTO JRITE(10T,907) FORMAT(//INPUT COLD SIDE INLET TEMPERATURE:') READ(IN,902) TCLDIN WRITE(IOT,906) FORMAT(/7'INPUT HOT SIDE INLET TEMPERATURE:') READ(IN,902) THOTIN CONSTANT TEMPERATURES FRONT END $\begin{array}{l} \operatorname{SET2}(1) = \operatorname{THOTIN} \\ \operatorname{SET2}(2) = \operatorname{TCLDIN} \end{array}$ 66667 8 11NIT = 250 = 2 = 1 = 3,8 VALK1 = CHOT VALK2 = CCLD = 1 = 1, 3TINIT = 100. 11 D0 20 L2(I) -FL4 (1) FL4 (2) FL4 (2) FL4 (4) CLELLUND CLELLUND CLELLUND 904 20 905 906 2 907 60 C υ C C 000 C 000 υ







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APPENDIX C NTU32C COMPUTER GENERATED INPUT ANALYZER PROGRAM

THIS IS PROGRAM NTU32C

C THIS IS PROGRAM NTU32C	C IT GENERATES AN INPUT FILE FOR THERMAL ANALYZER ' C I-3 EFFECTIVENESS-NTU RELATIONSHIP THAT IS NOT AN C OPEN LITERATURE (2C MEANING ONE PARALLEL PASS AN C COUNTERFLOW PASSES).	DIMENSION COEF(200,5), KCON(200,5), L1(8), L2(3), L3(6),	C CHARACTER *79 TITLE CHARACTER *12 FNAME	DATA IOT, IN, IPR, IWR/6,5,4,8/	C OPEN PRINTER OUTPUT FILE	C OPEN(IPR,FILE='PRN',STATUS='NEW',FORM='FORMATTED',IO IF(ICK.NE.0) WRITE(10T,920) 920 FORMAT('Trouble opening printer output file') C	917 FORMAT(//Tinput the title of this study - 79 columns & This title will appear',/, at the top of every p	918 FORMAT(Å79) TIS FORMAT(Å79)	901 FORMAT(/7,901) 901 FORMAT(/0,1NPUT HOT SIDE CAPACITY RATE:')	902 FORMAT(BN,F10.0)	903 FORMAT(//T,903) READ(IN,902) CCLD SIDE CAPACITY RATE:')	C WRITE(10T,904) 904 FORMAT(/71,902) U OVERALL HEAT TRANSFER COEFFICIENT:')	
	AL ANALYZER TO OBTAIN HAT IS NOT AVAILABLE IN LLEL PASS AND TWO	L2(3),L3(6),SET2(2),FL4(4)				ORMATTED',IOSTAT=ICK) file')	- 79 columns only. ', p of every printed påge',		((.	EFFICIENT:')	

WRITE(IOT,905) FORMAT(// INPUT TOTAL HEAT TRANSFER SURFACE:') READ(IN,902) SURFTO WRITE(IOT,907) FORMAT(//INPUT COLD SIDE INLET TEMPERATURE:') READ(IN,902) TCLDIN WRITE(107,906) FORMAT(//INPUT HOT SIDE INLET TEMPERATURE:') READ(IN,902) THOTIN E CHOT
E CCLD
CCLD
U*SURFTO/150.
= 125. CONSTANT TEMPERATURES **READY FOR INPUT SET 4** $\begin{array}{l} \text{SET2} \left\{ 1 \right\} = \text{THOTIN} \\ \text{SET2} \left\{ 2 \right\} = \text{TCLDIN} \end{array}$ FRONT END $\begin{array}{rcl} FL4\{1\} &=& 05\\ FL4\{2\} &=& .66667\\ FL4\{3\} &=& .8\\ FL4\{4\} &=& 12\\ L4 &=& 12 \end{array}$ = 200 = 1=3,8 = 0 $= \begin{bmatrix} I = 1, 3 \\ 0 \end{bmatrix}$ 20 D0 20 20 L2(I) VALK1 VALK2 VALK3 TINIT $L1 \{ 2 \} D0 \{ 10 \} D0 \{ 10 \} D1 \{ 1 \}$ ELLISSE CONTROLLESSE CONTROLLES NODE 1 905 10 906 907 C υ 000 υ 0000 υ υ U ບບບ



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OPEN(IWR,FILE='TVSSI',STATUS='NEW',FORM='FORMATTED',IOSTAT=ICK) IF(ICK.GT,0)WRITE(IOT,924) FORMAT(/, Trouble opening input file') 922 FORMAT(/, ENTER NAME OF INPUT FILE, INCLUDING DRIVE' & DESIGNATION:) READ(IN,923) FNAME I=1,3 I=1,6 1_FL4(2),L4,FL4(3),FL4(4) DO 250 I=51 200 WRITE(IWR,913) KCON(I,1),KCON(I,2) WRITE(IWR,914) COEF(I,1),COEF(I,2) D0 200 I = 1 50 WRITE(IWR 913)(KCON(I,J),J=1,4) FORMAT(918) WRITE(IWR 914)(COEF(I,J),J=1,4) FORMAT(4F8.4) CONTINUE [†] ^{Al4} END OF DATA SETUP NOW CREATE INPUT FOR ANALYZER (L1(I),I=1,8) RITING FILE : RITING FILE : 9) TITLE CLOSE(IWR, IOSTAT=ICK) WRITE(IOT,925) FNAME WRITE(IOT,925) FORMAT(/, WRITING F. OPEN INPUT FILE 1X,91 a a a a a a a a a 135 KCON $\left(\begin{array}{c} I \\ I \end{array} \right)$ KCON $\left(\begin{array}{c} I \\ I \end{array} \right)$ COEF $\left\{ \begin{array}{c} I \\ I \end{array} \right\}$ 923 FORMAT FORM 924 919 925 925 912 908 913 914 200 250 911

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C 921 F(ICK NE.0) WRITE(IOT,921)EEX C 921 FORMAT(Trouble closing printer output file') STOP END

ρ APPENDIX

NTU32P COMPUTER GENERATED INPUT ANALYZER PROGRAM

THIS IS PROGRAM NTU32P

DIMENSION COEF(200,5),KCON(200,5),L1(8),L2(3),L3(6),SET2(2),FL4(4) 917 FQRMAT(/ Input the title of this study - 79 columns only ', This title will appear',/, at the top of every printed page' & of output:', 918 FORMAT(Å79) IT GENERATES AN INPUT FILE FOR THERMAL ANALYZER TO OBTAIN 1-3 EFFECTIVENESS-NTU RELATIONSHIP THAT IS NOT AVAILABLE IN OPEN LITERATURE (2P MEANING TWO PARALLEL PASSES AND ONE COUNTERFLOW PASS). OPEN(IPR,FILE='PRN',STATUS='NEW',FORM='FORMATTED',IOSTAT=ICK) IF(ICK,NE.0) WRITE(10T,920) FORMAT(' Trouble opening printer output file') WRITE(IOT,904) FORMAT(// INPUT OVERALL HEAT TRANSFER COEFFICIENT:' READ(IN,902) U WRITE(IOT,901) FORMAT(//INPUT HOT SIDE CAPACITY RATE:') READ(IN 902) CHOT FORMAT(BN,F10.0) WRITE(10T,903) FORMAT(//INPUT COLD SIDE CAPACITY RATE: READ(1N,902) CCLD DATA IOT, IN, IPR, IWR/6,5,4,8/ OPEN PRINTER OUTPUT FILE CHARACTER *79 TITLE CHARACTER *12 FNAME 902 920 901 903 904 000000000 υ 000 C C C C C

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(107,905) (// INPUT TOTAL HEAT TRANSFER SURFACE:') IN,902) SURFTO WRITE(107,907) FORMAT(//INPUT COLD SIDE INLET TEMPERATURE:' READ(1N,902) TCLDIN WRITE(IOT,906) FORMAT(//INPUT HOT SIDE INLET TEMPERATURE:' READ(IN,902) THOTIN = CHOT = CCLD = U*SURFTO/150. = 125. CONSTANT TEMPERATURES **READY FOR INPUT SET 4** SET2(1) = THOTIN SET2(2) = TCLDIN FRONT END = :056667 = :66667 = :8 12 = TINIT = 200 = 2 = 1=3,8 = 1 = 1, 3300 6040 WRITE(FORMAT READ(I VALK1 VALK2 VALK2 VALK3 TINIT FL4 (1) FL4 (2) FL4 (2) FL4 (4) L4 = 1 D0 20 L2(I) NODE 1 20 905 10 906 907 C 0000 υ C 000 υ C ပပပ

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OPEN(IWR,FILE=FNAME,STATUS='NEW',FORM='FORMATTED',IOSTAT=ICK) IF(ICK,GT,0)WRITE(IOT,924) FORMAT(/, Trouble opening input file') 922 WRITE(IOT,922) 922 FORMAT(/ Enter name of input file, including drive' & DESIGNATION:) 923 FORMAT(A12) 923 FORMAT(A12) 4,FL4(3),FL4(4) CLOSE(IWR,IOSTAT=ICK) IF(ICK.NE.0) WRITE(IOT,921)EEX FORMAT(' Trouble closing printer output file DO 250 I=51 200 WRITE(IWR,913) KCON(I,1);KCON(I,2) WRITE(IWR,914) COEF(I,1);COEF(I,2) CONTINUE D0 200 I = 1 50 WRITE(IWR 913)(KCON(I,J),J=1,4) FORMAT(918) WRITE(IWR 914)(COEF(I,J),J=1,4) FORMAT(4F8.4) CONTINUE OT,925) FNAME WR,919 1X,919 1X,379 1X,379 1X,908 (L1(I),I=1,8) 4(2)₁ ET2(2) END OF DATA SETUP NOW CREATE INPUT FOR ANALYZER , I=1, = 10*K = VALK OPEN INPUT FILE WRITE(IOT, 925) FORMAT(/, Wr1 WRITE(IW, 919) 080 KCON(I,2) COEF(I,2) 120 COEF(I,2) WRITE FORMA FORMA 924 919 925 912 250 908 913 914 200 911 921 0000 C C 000 C ပ C



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APPENDIX E SAMPLE OUTPUT FROM NTU14 COMPUTER INPUT ANALYZER PROGRAM

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APPENDIX F

MODIFIED SECTIONS OF THERMAL ANALYZER TO RUN ON BATCH SYSTEM

QHARE JOB (2323,0267), TYCCOUNT, CLASS=C MAIN ORE WPECWILZ, PARM.LKED LISTTMAD FORT:SYSIN DD., THE WARTLY, PARM.LKED LISTTMAD THIS IS TYCCOUNT WHICH T2075 00511 <<<<<<<<><<<<>><<<<>><<<>><<<>	<pre>INPUT IS WRITTEN ON UNIT 3 BEFORE NORM DELETE INPUT LIST, INSERT DUMMY MENT ************************************</pre>
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THERE AKE ERRORS IN THE ONS AND RETURN TO ERR) -TOLD(I)3) GO TO 120 , ALPHA, BETA, TOLD, TOLD1, TOLD2, TD 1,NTCOEF) (I)H RTAG = 1 TITLE) K=37,LOCH)) * Lp1(I) = 0.0Α, ERI 107 GO TO 110 - WCOEF*T1 - WCOEF CASBTU K=1,NP1 THIS LL SK FOR BETA NP1, H) .40,] F IER.NE.0 AT NPUT; TVSSI WI TATEMENT 1000 (IER.EQ.0) ro 1000 ro 1000 $\left\{ \begin{array}{c} = & A \left\{ NP1 \\ = & A \left\{ NOD \right\} \\ UE \end{array} \right\}$ $(2)^{-NOCASE}$ S Z $(\bar{2})^{A}$ н CHOST ENDFILE 2 REWIND 1 REWIND 2 NODEI TOEI G NOD 0 URITE GO TC 0 UL FON N ÅL 05 4 õ L. -36 z 75 80 850 1005001 100500 100500 * * 110 120 140 145 * 130 135 -}¢

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ĪÕNŠ''/' BETAS'// 2006.4) EPTABLE' FIX DECK AND RESUBMIT AN GO TO 175 SSIBLE NOPE NUMBER - YOU SPEC NODE I SODE TO A NODE - I ALID'NODE - YOU SPECIFIED / DE, 15,5X,TO A NODE I5/15X 14, ... CONSTANT TEMPERATURES 5 $\begin{array}{c} \left(\overset{\circ}{I} \overset{\circ}$ 6 E - - YOU HEATERS') MPĔRĂTI TMPHT) D NODI s NÕŤ 155 0 WAT (1518) RMAT (1518) INTERACTION FROM NO INTERACTION FROM NO INTERACTION FROM NO INTERACTION FROM NO BUT THERE ARE ONLY BUT THERE ARE ONLY BUT THERE ARE ONLY conthe(I); NN INTERACTION FROM BUT THERE ARE ON RMAT (/IX * * * INTERACTION FROM BUT THERE ARE ON RMAT (/IX * * 00 INTERACTION FROM BUT THERE ARE ON RMAT (/IX * * * * NOHTR **Ř**Ê ĂRÊ ΥE 0A4 814 vr(1518) မိုထိုထ ^ NEB 80E FORMAT (FORMAT (FORMAT (FORMAT FORMAT WRIT GO T WRIT WRIT 590 FOR URI TF 600ŢFQĔ 004 0000000 610_FQ EO 160 2005 2005 2005 150 175 553 620 50

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THER CALCULATIONS FOR THIS PROBLEM (SET 7)') X' TO NODE' THERE ARE ONLY /15X, .,/15X VES ERR Ô.) GO TO 10 . LE. ERR) GO TO 50 - ALPHA) BETA(I) = -BETA(I)*ALPHA/GAMMA , TNM2(315) , BETA(315) BETA, TN, TNMI, TNM2, NCF OF , T Å²(Ι) E-06) T12 = SIGN(IE-06,T12 D A CONDUCTANC TEMP-DEPENDENT , SAVE(49770) ECTFIED A CON TED CONDUN PECTFIED 11 GOTO 50 ALPHA BETA(I) = (A-H,0-Z) LOCS(316) $\frac{0-2}{315}$ $\frac{I}{READ} \begin{bmatrix} +1\\ 2 \end{bmatrix} (EL(K), K=1, NP1)$ 9F8.2 SUBROUTIN DIMENSI LOCS (1 NM1 = 0 I = 0 **RETURN** END SUBROI CONT STOP END GAM 04 04 6 670 680 680 640 650 660 10 630 10 50

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OK3+9) * FRAC + HTR(LOK3+4 LOK3+9))*FRAC + HTR(HTR, CASBTU, NOCONT, TOLD, NODCFH) SIGN(1E-06, HTRL3 TOLD(315) 2+4)) 5,5,15 2+9 HTR(LOK2+14) (NL, FNAME) A-H, O-Z) Ğ ŘÍ=5,8 |= LoK2+K1 |TEMP - HTR(LOK3)) 17,17,16 = HTR(LOK2+13) CASBTU + HTR(LOK2+18) LOK2+3) 5,70 24,24,43 I, NOHTRS HTR 0 LOK) LOK) 25 = 0 SUBROU **RETURI** END DIMEN CASBT DO 25 \ŝàħ SUBRO ITR(ASB DIME LOK Y **NO** Š 43 45 0000 15 16 24 25 10 0 Z 17

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///LKED.SYSLMOD DD DISP=SHR,DSNAME=MSS.S2323.LOADLIB //LKED.SYSIN DD * ...NAME TVCOUNT(R) IF (LINCNT - 56) 40,40,30 NPAGE = NPAGE + 1 WRITE (8,50) FNAME,NPAGE LINCNT = NL RETURN FORMAT (1H1,20X,20A4,8X,9HPAGE NO. ,13/) END CNT = LINCNT + NL (LINCNT - 56) 40,40,30 E = NPAGE + 1 E (8,50) FNAME,NPAGE 30 200 ÷ ×

<u>APPENDIX</u> <u>G</u> MODIFIED NTU14 PROGRAM NTU14BC USED TO RUN ON BATCH SYSTEM

SYSTAT SYSTAT
TINE TINE



COEF(I,1) = VALK1 DOB(0 I] = VALK1 COEF(I,II) = VALK3 CONTINUE CONTINUE E 51	KCON {51,1} = 3025 KCON {51,2} = 14 COEF {51,1} = VALK2 COEF {51,2} = VALK2	DES 52 TO 250	D0_120_1 = 52,250 K0_17_1]F(I.GT.100) GO TO 122 J [1 - 50	G0 T0 135 IF(I.GT.150) G0 T0 124	GO_TÔ 135 IF(I.GT.200) GO TO 126 J I I - 150	GO TO 135 CONTINUE L = I - 200 M = 7*1 - 1	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	$\begin{array}{rcl} & \begin{array}{c} & \begin{array}{c} & \begin{array}{c} & \end{array} \\ & \end{array} \\ & \begin{array}{c} & \end{array} \\ & \begin{array}{c} & \end{array} \\ & \begin{array}{c} & \end{array} \\ & \end{array} \\ & \begin{array}{c} & \end{array} \\ & \begin{array}{c} & \end{array} \\ & \end{array} \\ & \end{array} \\ & \begin{array}{c} & \end{array} \\ & \end{array} \\ & \end{array} \\ & \begin{array}{c} & \end{array} \\ & \begin{array}{c} & \end{array} \\ \end{array} \\ & \end{array} \\ \end{array} \\$	D OF DATA SETUP W CREATE INPUT FOR ANALYZER	WRITE(2,919) TITLE(P,1) FORMAT(1XA25),(1) T-1 8)	FORMAT(6)749/(LL1(1),1-1,0) FORMAT(9]49 WEITE(2,908)(L2(]),1=1,3)	WRITE(2,9U8)(L3(L),L=L,0)
80 75 NOD		ON			122	 124	126	5	120	EN	616	908	
5 3 5 3	ra r	· 16.16 ·	1						1	יר זר זר	. 3		



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\$\${2E=6160)
\$\$\$IB, :{że=6160) ;§IE, 12E=6160) SSIC, 2812E=6160) V\$\$1D, \$\${\$£=6160)
\$\$\$16, ;{<u>2</u>E=6160) ;§IH, <u>†że</u>=6160) șII, (ŻE=6160) SIF, ZE=6160) ÍŻE=6160) SIK, 200. ŻE=6160) 200. STOAD လိုလ် , CLASS=P 15 15 TUE VERSES (NEW PASS) (NEW P Ľ=¤ð`Ē CYL, DSN CL = 80CYL CYL 'NTU14bL' 0.08333 TVSSIA 0.41667 TVSSIB =LOCA ECFM=FB. SDA, SP/ CFM=FB. NEW YSD, CFN YSD7 ECEN SD CONDUCTION CONTRACT C 25.0 NTU14 25.0 NTU14 ///GO.FT01F001 DD NTU=0.05 k=0.10 h NTU=0.250 k=0.10 h GO. TVSSIB GO. TVSSIC GO. TVSSID GO. TVSSIE GO. TVSSIF GO. TVSSIG GO. TVSSIH GO.TVSSIJ GO. TVSSIK GO. TVSSII 00 NTU=0NTU=0

APPENDIX H

NTU14BL LIBRARY BATCH PROGRAM

100. 100. 100. 100. 100. 100. 100.	SSIA SSIB SSIB SSIC SSIC
200. 200. 200. 200. 200. 200. 200.	2323.L0ADI 4, DSN=&TV 2323.L0ADI 4, DSN=&TV 2323.L0ADI 6, DSN=&TV 4, DSN=&TV
	LIBE MSS CYL, HILL CYL, HILL C
0.83333 1.25000 1.25000 1.25000 1.25000 1.25000 2.50000 3.33333 3.33333 4.16667 4.105811 5.41667 105811 5.41667 105811 5.41667	SYSDA, SPACE= SYSDA, SPACE= SY
0 255.04 0 255.04 0 255.014 0	A H H H H H H H H H H H H H H H H H H H
U=0.250 k=0.1 U=0.250 k=0.1 U=1.250 k=0.1 U=1.250 k=0.1 U=1.250 k=0.1 U=2.250 k=0.1 U=3.250 k=0.1 U=3.250 k=0.1 U=3.250 k=0.1 U=3.250 k=0.1	STEPA Solution Soluti

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SN=&TVSSID 23.LOADLIB')SN=&TVSSIE 23.LOADLIB'	SN=&TVSSIF 23.LOADLIB'	SN=&TVSSIG 23.LOADLIB'	SN=&TVSSIH 3.LOADLIB'	SN=&TVSSII 23.LOADLIB'
NIT=SYSDA,1 EBA B='MSS:S232 YL, [1, 1]} YL, [1, 1]}	NIT=SYSDA, 1 FBA B= MSS, 8233 VL, {1, 1 VL, {1, 1}} YL, {1, 1}}	NIT=SYSDA, I FBA B= MSS, S233 VL, (1, 1, 1) VL, (1, 1, 1) VL, (1, 1, 1)	WIT=SYSDA, I FBA B= MSS, S232 VL, [1, 1] VL, [1, 1] VL, [1, 1]	WIT = SYSDA, I FBA B= MSS, S232 VL, [1, 1] VL, [1, 1] VL, [1, 1] VL, [1, 1]	WIT'= SYSDA, I FBA B= MSS, S2 FL, [1, 1] YL, [1, 1] YL, [1, 1] YL, [1, 1]
DELETE), UI DELETE), UI CVCOUNT, LI V SPACE (CI A, SPACE (CI A, SPACE (CI A, SPACE (CI A, SPACE (CI A), SPACE (CI	DELETE), UI FVCCOUNT, LITI FVCCOUNT, LITI A, SPACE= CC A, SPACE= CC A, SPACE= CC	DELETE), UI FVCCUNT, LII FVCCUNT, LII A, SPACE= C A, SPACE= C A, SPACE= C C	DELETE) CCB=RECFM=U CCCB=RECFM=	DELETE), UI CCBERECEMENT V SCOUNT, LIII V SPACE CC V SPACE CC V SPACE CC	DELETE) CCB=RECEMUN CVCB CVCB=RECEMUN CVCB CVCB=RECEMUN CVCB CVCB CVCB CVCB CVCB CVCB CVCB CVC
DUMMY DISP=(OLD SYSOUT=A,I TVG PROG=1 UNIT=SYSDA UNIT=SYSDA UNIT=SYSDA	DUCHAT SUSSECT	DUCHT DUCHT DUCE DUC DUC DUC DUC DUC DUC DUC DUC DUC DUC	DUMMY DISP=(OLD) TVSOP(CALD) TVSOPCG=1 UNIT=SYSDA UNIT=SYSDA UNIT=SYSDA	DUMMY DISP=(OLD) TVG_PROG=1 UNIT=SYSDA UNIT=SYSDA UNIT=SYSDA	DUMMY DISP=(OLD IVSCPR06=1 UNIT=SYSDA UNIT=SYSDA UNIT=SYSDA UNIT=SYSDA
. FT03F00 . FT03F00 . FT08F00 . FT01F00 . FT01F00 . FT02F00 . FT09F00		EFT02F00 EFT02F00 EFT01F00 ETT02F00 ETT02F00 ETT02F00	FT04F00 FT04F00 FT01F00 FT01F00 FT02F00 FT102F00 FT102F00	. FI04F00 . F104F00 . F108F00 . F108F00 . F101F00 . F102F00 . F109F00	FT103F00 FT103F00 FT108F00 FT108F00 FT101F00 FT102F000 FT109F000
				000H0000	

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APPENDIX I

MODIFIED NTU32C PROGRAM NTU32CC USED TO RUN ON BATCH SYSTEM

OBTAIN	TWO	T2(2),FL4(4)			TIN(0,5),TCL						
MAI. ANALYZER TO	THAT IS NOT AVA	o ¹² (3), ¹³ (6), ^{SE}			SURFTO(0,4),THO			~			
132CC',CLASS=G TEM=SY2' LIST.MAP' 132C 11T FILE FOR THEF	ITU RELATIONSHIP MEANING ONE PAR	CON(200,5),L1(8) 22),FNAME(22,22)			crb(0,2),U(0,3),	0.0) FNAME(P,2)		FORM= ' FORMATTED'	,4)/150.		
B (2323 0267) 'NTU G=NPGVM1 2323P SY RTVCL, PARM.LKED= 1 N DD . PROGRAM NTU HIS IS PROGRAM NTU	- 3 EFFECTIVENESS-1 PEN LITERATURE (20 OUNTERFLOW PASSES	ER 0, P SION COEF(200 5), (22,22), CCLD(22,22) N(22,22), TITLE(22	CTER *25 TITLE CTER *25 FNAME	0=1,21,2	1,900) CHOT(0,1),0	T(2F10.0F10.5,3F) T(2A25) T(2A25)	OUTPUT FILE	2,FILE=FNAME(P,2)	$= CHOT \{0, 1\} \\= CCLD \{0, 2\} \\= U(0, 3) & SURFTO(0) \\= 125.$	FRONT END	= 200 = 2 T=3 8
// WARE JO MAIN OR EXEC FO FORT.SYSII	4-00	2†CLDII	CHARA	D0 7 0	READ	900 FORMA	OPEN (OPEN(:	VALK1 VALK2 VALK3 VALK3 TINIT		$ \begin{array}{c} L1 \\ L1 \\ L1 \\ 2 \\ 10 \\ 1 \\ 1 \\ 1 \\ 1 \\ 1 \\ 1 \\ 1 \\ 1 \\ $



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FL4(3),FL4(4) **¢ČŎN(I,J),J=1,4**) END OF DATA SETUP NOW CREATE INPUT FOR ANALYZER (1(1), 1=1.8)TITLE(P,1) 00) GO TO 122 50) GO TO 124 = 52,200 = VALK3 NODES 52 TO 200 -135 150 (616)н н н KCON [51,1] KCON [51,2] COEF [51,2] COEF [51,2] 5 2*L . 5 80 CONTINUE 75 CONTINUE GT.] $\begin{array}{c} D0 & 120 \\ K &= I \\ IF(I.GT \end{array}$ × WRITE (FORMAT DO 20 WRITE FORMA KCON (COEF NODE 51 G0 T FOR н 135 120 919 124 912 913 908 122 911 ပပပ ບບບ 0000 C

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///LKED.SYSLMOD DD DISP=SHR,DSNAME=MSS.S2323.LOADLIB //LKED.SYSIN DD * ... NAME COUNTER(R) 200 KCON(I,1),KCON(I,2) COEF(I,1),COEF(I,2) WRITE(2,914)(COEF(I,J),J=1,4) FORMAT(4F8.4) CONTINUE DO 250 I=51 WRITE(2,913) WRITE(2,914) CONTINUÈ CONTINUE STOP END 914 200 250 ~ *__ * ပ υ

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23.LOADLIB' SIA. tże=6160) ssie, {\$ { 2 } { 2 {\$ 12E=6160)
\$\$\$ID, tże=6160) StF, (†2E=6160) SSIG, [ŻE=6160) SIH, \$\${2E=6160)
\$\$\$11, ZE=6160) IK, 200. E=6160) ŽE=6160) 200. NTU32CL', CLASS=P 255 252 15 15 CVL=80 CVL=80 CVL=80 CVL=80 DSN DSN L=8(DSI -11 60 3333 SSTA 667 SSIB =LOCA E FB. DUCULIANCE CERT DUCULIANCE DUCULI I00.0 COUNTER COUNTER //GO.FT01F001 DD 250 NTU=0.05 R=0.40 C R=0.40 GO. TVSSIB GO. TVSSIC GO. TVSSID GO. TVSSIE GO. TVSSIF GO. TVSSIG GO. TVSSIH GO. TVSSII GO.TVSSIJ GO. TVSSIK NTU=0.25 00

APPENDIX J

NTU32CL LIBRARY BATCH PROGRAM

100. 100. 100. 100. 100. 100. 100. 100.	IB' SIA SIB SIB SIB SIC SIC
200. 200. 200. 200. 200. 200. 200.	323.LOADL , DSN=&TVS 323.LOADL , DSN=&TVS 323.LOADL 323.LOADL , DSN=&TVS 323.LOADL
15. 15. 15. 15.	CYL, ([1,1]) CYL, ([1,1]) CY
0. R=0.40 C00NTER 5.0005510 0. 0. 0. 0. 0. 0. 0. 0. 0. 0	ExecFould2F001DDUNITT2F001DDUNITT2F001DDUNITT3F001DDUNITT3F001DDUNITT3F001DDUNITT3F001DDUNITT3F001DDUNITT3F001DDUNITT3F001DDUNITT3F001DDUNITT3F001DDUNITT3F001DDUNITT3F001DDUNITT3F001DDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT3F001DDDDUNITT
$\begin{array}{c} \mathbf{T} \mathbf{T} \mathbf{U} = 0 \\ \mathbf{U} \mathbf{U} \mathbf{U} \mathbf{U} \\ \mathbf{U} $	

DĪŠP=(OLD, DELETE), UNIT=SYSDA, DSN=&TVSSIE SYSOUT=A, DCB=RECFM=FBA TVG_PROG=TVCOUNT, LIB='MSS, S2323.LOADLIB' UNIT=SYSDA, SPACE={CYL, {1,1}} UNIT=SYSDA, SPACE={CYL, {1,1}} UNIT=SYSDA, SPACE={CYL, {1,1}} ESP= (OLD DELETE) UNIT=SYSDA, DSN=&TVSSIG CSOUT=A DCB=RECFM=FBA CG PROG=TVCOUNT, LIB='MSS ; S2323.LOADLIB' VIT=SYSDA, SPACE={CYL, {1, 1}} ISP=ISP=VSOUT=ADCB=RECFM=FBAVGPROG=TVCOUNTVIT=SYSDASPACE=CYLVIT=SYSDASPACE=CYLVIT=SYSDASPACE=CYLVIT=SYSDASPACE=CYLVIT=SYSDASPACE=CYLVIT=SYSDASPACE=CYLVIT=SYSDASPACE=CYLVIT=SYSDASPACE=CYLVIT=SYSDASPACE=CYLVIT=SYSDASPACE=CYLVIT=SYSDASPACE=CYLIIVIT=SYSDASPACE=CYLVIT=SYSDASPACE=CYLVITSYSDASPACECYLVITSYSDASPACECYLVITSYSDASPACESPACESPACECYLVITSPACE</t UNIT=SYSDA, DSN=&TVSSIF I=FBA JEB= 'MSS, S2323.LOADI.TR' UNIT=SYSDA,DSN=&TVSSID =FBA DISP= (OLD, DELETE), UNIT=SYSDA, DSN=&TVSSII SYSOUT=A, DCB=RECFM=FBA CTVG, PROG=TVCOUNT, LIB='MSS, S2323. LOADLIB' UNIT=SYSDA, SPACE= (CYL, (1,1) UNIT=SYSDA, SPACE= (CYL, (1,1)) UNIT=SYSDA, SPACE= (CYL, (1,1)) 323.LOADLIB' (ČŸL, ((č<u>v</u>l; (ĵ ŝ ULSP=(OLD, DELETE), UN SYSOUT=A, DCB=RECFM=F (TVG, PROG=TVCOUNT, LIB UNIT=SYSDA, SPACE=(CYI UNIT=SYSDA, SPACE=(CYI UNIT=SYSDA, SPACE=(CYI UNIT=SYSDA, SPACE=(CYL UNIT=SYSDA, SPACE=(CYL 20 SYSOUTESYSDA, DCBERECFME UNITESYSDA, SPACE= ר= ד= גיי L DD UN C DD C DD C T DD SL FORTVC PORTVOR 0000 08F00 8F00 02F00 02F00 09F00 0F0 4 08] 08 08 σ œ 00 STEP. TEP 0.1 Ô. V °, Q o. , Ö , S • • -. • . • 001000 ບັດບັ

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APPENDIX K

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MODIFIED NTU32P PROGRAM NTU32PC USED TO RUN ON BATCH SYSTEM

OH32PC JOB (2323,0267), 'NTU32PC', CLASS=G *MAIN ORG=NPGVM1.2323P, SYSTEM=SY2 * EXEC FORTVCL, PARM.LKED='LIST,MAP' FORT.SYSIN DD * PROGRAM NTU32P	IT GENERATES AN INPUT FILE FOR THERMAL ANALYZER TO OBTAIN 1-3 EFFECTIVENESS-NTU RELATIONSHIP THAT IS NOT AVAILABLE IN OPEN LITERATURE (2P MEANING TWO PARALLEL PASSES AND ONE COUNTERFLOW PASS).	INTEGER 0.P DIMENSION COFF(200,5),KCON(200,5),L1(8),L2(3),L3(6),SET2(2),FL4(4) 1,CHOT(22,22),CCLD(22,22),U(22,22),SURFTO(22,22),THOTIN(22,22), 2†CLDIN(22,22),TITLE(22,22),FNAME(22,22)	CHARACTER *25 TITLE CHARACTER *25 FNAME	DO 7 0=1,21,2 P=0+1 READ(1,900) CHOT(0,1),CCLD(0,2),U(0,3),SURFTO(0,4),THOTIN(0,5),TCL	900 FORMAT(2F10.0F10.5;3F10.0) READ(1918) TITLE(P;1),FNAME(P,2) 318 FORMAT(2A25)	OPEN OUTPUT FILE	OPEN(2,FILE=FNAME(P,2),FORM='FORMATTED')	VALKI = CHOT(0,1) VALK2 = CCLD(0,2) VALK3 = U(0,3)*SURFTO(0,4)/150. TINIT = 125.	FRONT END	$ \begin{array}{l} L1(1) &= 200 \\ L1(2) &= 2 \\ D0 & 10 \\ 1 &= 3, 8 \end{array} $
	000000	ט כ	ט נ	د		ວບເ	с c	د	000	C



L4 FL4(3), FL4(4) $\begin{pmatrix} 2 & \frac{1}{9} & \frac{1}{3} \\ \frac{1}{9} & \frac{1}{2} \end{pmatrix} \begin{pmatrix} 1 & 50 \\ 1 & 1 \end{pmatrix} \begin{pmatrix} 1 & 1 \\ 2 & 0 \end{pmatrix} \begin{pmatrix} 1 & 1 \\ 1 & 1 \end{pmatrix} \begin{pmatrix} 1 & 1 \\ 1 & 1 \end{pmatrix}$ [4)(COEF(I,J),J=1,4) END OF DATA SETUP NOW CREATE INPUT FOR ANALYZER Ĺl(Ι),Ι=1,8) 150) GO TO 124 = 52,200 = VALK3 3025 14 VALK2 VALK3 NODES 52 TO 200 KCON (51,1) KCON (51,2) COEF (51,2) COEF (51,2) 80 CONTINUE 75 CONTINUE × WRITE (FORMAT KCOON COCEF ORM NODE 51 120 616 122 124 135 908 912 913 911 0000 υ ပပပ ပပပ

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APPENDIX L NTU32PL LIBRARY BATCH PROGRAM

■ SOUNDANDANDANDANDANDANDANDANDANDANDANDANDAN	15.
 N N N N N N N N N N N N N N N N N N N	R 1VSSTB
	60 COUNTE
QH2PL JOB WAIN ORG EXECTFOR GO. TVSSIL GO. TVSSIN GO. TVSSIN GO. TVSSIN GO. TVSSIP GO. TVSSIP GO. TVSSIS GO. TVSSIS GO. TVSSIS GO. TVSSIS GO. TVSSIS GO. TVSSIS GO. TVSSIV	U=U, V3 K-V. 11=0, 25 R=0.

100. 100.

100. 100. 100. 100. 100. 100. 100.	DLIB' VSSIL VSSIL VSSIM VSSIM VSSIN VSSIN VSSIN
200. 200. 200. 200. 200. 200. 200.	<pre>\$2323.LOA DA,DSN=&T DA,DSN=&T DA,DSN=&T DA,DSN=&T DA,DSN=&T DA,DSN=&T DA,DSN=&T DA,DSN=&T DA,DSN=&T DA,DSN=&T</pre>
15. 15. 15. 15.	
R 7.50000 R 7.500000 R 7.50000 R 7.500000 R 7.50000 R 7.500000 R 7.500000 R 7.5000000 R 7.5000000 R 7.5000000 R 7.50000000 R 7.5000000000 R 7.5000000000000000000000000000000000000	SYSDA, SPACEE SYSDA, SPACEE
. 60 150.0 . 70 1	A HELE COLUCTION COLUCTICO COLUCTION COLUCTICA
U=0.550 k=0 U=1.550 k=0 U=1.550 k=0 U=1.550 k=0 U=1.550 k=0 U=2.550 k=0 U=2.550 k=0 U=3.250 k=0 U=3.250 k=0 U=3.250 k=0	CONTRACTION CONTRA

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E), UNIT=SYSDA, DSN=&TVSSIT CFM=FBA S2323.LOADLIB MSS 11 LIB DISPE(OLD, DELETE) SYSOUT=A, DCB=RECFN SYSOUT=A, DCB=RECFN INIT=SYSDA, SPACE= UNIT=SYSDA, SPACE= UNIT=SYSDA, SPACE= UNIT=SYSDA, SPACE= -11 ī н Ť08F001 Ť08F001 EXEC FT01F0 FT02F00 T102F00 T03F000 T04F0001 T08F001 01F001 Construction of the second state of the second 000 • FT01F • FT02FC • T09F0 • T19F00 002F0 002F0 004F0 004F0 004F0 004F0 004F0 004F0 007F0 3F0 . FT02F FT01 . 0 TEH TEH . •

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DD DUMMY	DD DISP=(OLD, DELETE), UNIT=SYSDA, DSN=&TVSSIU	DD SYSOUT=A.DCB=RECFM=FBA	FORTVG, PROG=TVCOUNT, LIB='MSS. \$2323. LOADLIB'	DD UNIT=SYSDA,SPACE±(CYL,(1,1))	DD UNIT=SYSDA(SPACE=(CYL((1,1))	DD UNIT=SYSDA(SPACE=(CYL((1,1))	DD UNIT=SYSDA(SPACE=(CYL((1,1))	DD DUMMY	DD DISP=(OLD, DELETE), UNIT=SYSDA, DSN=&TVSSIV	ϦϽͺϚϒϚΛΙΙϮ<u>;</u>ΔͺϯϹ ϷΞϷϝϹϷ⋈ΞϷϷΔ
//G0.FT03F001	//GO.FT04F001	//GO.FT08F001	//STEPV EXEC	760.FT01F001	//G0.FT02F001	//GO.FT09F001	//GO.FT10F001	//GO.FT03F001	//G0.FT04F001	
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APPENDIX M

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Figure M.1 1-3:2C Effectiveness vs. N_{tu} over Range of R from 0.01 to 1.0



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Figure M.2 1-3:2C Effectiveness vs. N_{tu} over Range of R from 0.01 to 0.10



Figure M.3 1-3:2C Effectiveness vs. N_{tu} over Range of R from 0.11 to 0.2

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Figure M.4 1-3:2C Effectiveness vs. N_{tu} over Range of R from 0.21 to 0.3



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Figure M.5 1-3:2C Effectiveness vs. N over Range of R from 0.31 to 0.4



Figure M.6 1-3:2C Effectiveness vs. N over Range of R from 0.41 to 0.5

A- 0.00 1 3.26 1.00 2.76 2.60 EFFECTIVENESS VS. NT 2 OUT OF 3 PASSES IN COUNTER FLOW FOR R VARING BY .01 FROM .61 TO 0.6 2.25 **8**.8 SIN 1.76 8 1.26 8 0.76 8.0 2.0 5 . 3 3 **EFFECTIVENE88**

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Figure M.7 1-3:2C Effectiveness vs. N_{tu} over Range of R from 0.51 to 0.6



Figure M.8 1-3:2C Effectiveness vs. N over Range of R from 0.61 to 0.7



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Figure M.10 1-3:2C Effectiveness vs. Ntu over Range of R from 0.81 to 0.9



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Figure M.11 1-3:2C Effectiveness vs. N over Range of R from 0.9 to 1.0


1-3:2P EFFECTIVENESS VS. ${\tt N}_{\tt tu}$ GRAPHS AT VARIOUS R VALUES

APPENDIX N

Figure N.1 1-3:2P Effectiveness vs. N_{tu} over Range of R from 0.01 to 1.0



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Figure N.2 1-3:2P Effectiveness vs. N_{tu} over Range of R from 0.01 to 0.1



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Figure N.3 1-3:2P Effectiveness vs. N over Range of R from 0.11 to 0.2



S. L. L. L. L. L. L.

Figure N.4 1-3:2P Effectiveness vs. N_{tu} over Range of R from 0.21 to 0.3



S. Carrier

Figure N.5 1-3:2P Effectiveness vs. N_{tu} over Range of R from 0.31 to 0.4



gure N.6 1-3:2P Effectiveness vs. N over Range of R from 0.41 to 0.5

1979 3.26 9.0 2.76 2.60 2 OUT OF 3 PASSES IN PARALLEL FL FOR R VARING FROM ... 51 to 0.6 2.26 5.00 B 1.75 2 1.26 8 0.76 0.50 0.6 3 2 3 EFFECTIVENES8

Figure N.7 1-3:2P Effectiveness vs. N over Range of R from 0.51 to 0.6



a stand a stand

2.2

Figure N.8 1-3:2P Effectiveness vs. N over Range of R from 0.61 to 0.7



Figure N.9 1-3:2P Effectiveness vs. N_{tu} over Range of R from 0.71 to 0.8



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Figure N.10 1-3:2P Effectiveness vs. N over Range of R from 0.81 to 0.9



Figure N.11 1-3:2P Effectiveness vs. N over Range of R from 0.91 to 1.0

APPENDIX O

SAMPLE DISSPLAY PROGRAM USED FOR GRAPHING DATA

X4(I),Y4(I) X5(I),Y5(I)	X6(I),Y6(I)	X7(I),Y7(I)	X8(I),Y8(I)	(I)6X,(I)6X X10(I).V10(X	X11(I),Y11(I	X12(I),Y12(I	X13(I),Y13(I	X14(I),Y14(I	X15(I),Y15(I	X16(I),Y16(I	X17(I),Y17(I	X18(I),Y18(I	X19(I),Y19(I	
READ (24,1000) CONTINUE DO 60 I=1 11 READ (25,1000)	CONTINUE DO 70 I=1 11 READ (26,1000) CONTINUE	$\begin{array}{c} D0 & 80 & 1 = 1 \\ READ & (27, 1000) \\ CONTINUE \\ D0 & 90 & 7 = 1 \\ 11 \end{array}$	READ (28,1000) CONTINUE D0100 I=1.11	READ (29,1000) CONTINUE DO 110 I=1 11 READ (30 1000)	CONTINUE, 1000) DO120 I=1 11 READ (31,1000)	CUNTINUE DO 130 I=1 11 READ (32,1000)	DO 140 I=1 11 READ (33,1000)	DO 150 I=1 11 READ (34,1000) CONTINUE	DO 160 I=1 11 READ (35,1000) CONTINUE	DO 170 I=1,11 READ (36,1000) CONTINUE	$\begin{array}{c} \overbrace{0}{0} \overbrace{1}{1} \overbrace{8}{0} \overbrace{1}{1} = 1 \\ FEAD (37, 1000) \\ CONTINUE \end{array}$	DO 190 110 11 READ (38, 1000) CONTINIE	DO 200 I=1 11 READ (39,1000)	DO 210 I=1,11
0		0	0	00	.10	0Z.0	00	20	60	70	80	0 0		20

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## APPENDIX P

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# POLYNOMIAL REGRESSION CURVEFIT PROGRAM

PROGRAM POLYFIT	PURPOSE: $p_{X}^{O}$ , an NTH ORDER LEAST-SQUARES FIT OF THE INPUT VALUES $X_{X}^{O}$ , and $Y_{Y}^{O}$ .	INPUT: I, X, II) X.	DUTPUT: I) MEAN OF X. II) RANGE OF X. III) ORDER OF THE FIT IV) COEFFICIENTS IN DOUBLE PRECISION. V) R.M.S. ERROR OF THE RESIDUALS.	CALL: I) CURFIT (IORDER, NITEMS, X, Y, AA) II SOLVE(NR, C) III) FUNCTION AMAX(X, NITEMS) IV) FUNCTION AMIN(X, NITEMS)	TEGER I, NITEMS, MAXORD, J,K,L,M,N JUBLE PRECISION AA(40) MMENSION X(11),Y(11),YCOMP(11),ONE(11,2),TWO(11,2)	L. ZERO ALL VALUES THAT WILL BE USED TO COMPUTE SUMS.	15SUM = 0.0	2. INPUT THE DATA.	VIAGN=0 VRITE(6,100) PORMAT(2X, MAXIMUM ORDER OF THE FIT?')
PROGRAM	PURPOSE	INPUT:	OUTPUT:	CALL:	INTEGER DOUBLE P DIMENSIO COMMON I	1. ZER	RMSSUM =	2. INP	IDIAGN=0 WRITE(6 FORMAT(
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COMPUTE THE LEAST-SQUARES COEFFICIENTS BY CALLING "CURFIT()".
THE COEFFICIENTS ARE RETURNED IN "AA()" SUCH THAT:
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                        COMPUTE THE RESIDUALS AND THE R.M.S. ERROR OF THE RESIDUALS.
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                   CONTINUE \qquad RMSSUM = RMSSUM + (Y(I)-YCOMP(I))*(Y(I)-YCOMP(I))
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                    \sum_{\mathbf{Y} \in \mathsf{OMP}} \left\{ \mathbf{J} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{X} \left\{ \mathbf{I} \right\} = \left\{ \mathbf{X} \left\{ \mathbf{X} \left\{ \mathbf{X} \left\{ \mathbf{I} \right\}
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                              Y=AA(1)+AA(2)*X+...+AA(N)*(X**N)
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                           CALL CURFIT (MAXORD, NITEMS, X, Y, AA)
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                      RMS = SQRT(RMSSUM/FLOAT(NITEMS))
                                                                                                                                                                                                                                                                                                                                                                                                              ONE [1,1), TWO(1,2)
E [ 1, 3 ]
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                             PRINT OUT THE RESULTS
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                 EQ. 0.0
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FORMAT(2X,'')
READ(5,*),MAXORD
MAXORD=5
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WRITE(7,120)
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WRITE(7,108) I,XX(I),ZZ(I) NORMALIZE "X" AND "Z" BETWEEN "-1" AND "1". IF(IDIAGN, EQ. 3) WRITE(7,111) ICOEF, NITEMS
FORMAT(2X, 'CURFIT: ICOEF, NTIEMS = '2110) FORMAT(2X, 'CURFIT: XNORM, ZNORM = ', 3F15.5 FORMAT(2X,'CURFIT: I,XX,ZZ = ',I5,3F12.4) ÅŤŘÅ(100,100),AA(40) COMPUTE ALL NECESSARY SUMS. Z (NITEMS) DO 96 I = 1 NITEMS XX = X ZZ = X IF = X IFXMAX ABS(XMIN) ZMAX ICOEF = IORDER + 1 DIMENSION X(N DIMENSION XX( DIMENSION XX( DOUBLE PRECIS DIMENSION SUB DIMENSION SUB AMIN **6** 11 н ц 0.0 XXX XXX ZZZ ZZZ XNORM ZNORM XMAX XMIN ZMAX ZMIN ۲. 2. 101 C102 C102 

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CONTINUE SMATRX(J,ICOEF+1)=SMATRX(J,ICOEF+1)+SUBSUM(J)*ZZ(N) CONTINUE ', IF15.5) = SMATRX(J,I)+SUBSUM(I)*SUBSUM(J) NOW, INVERT THE MATRIX. (SOLVE THE SET OF SIMOLTANEOUS EQUATIONS.) "UNNORMALIZE" THE RETURNED COEFFICIENTS. FORMAT(2X,'CURFIT: SMATRX(',I3,',',I3,') = FORMAT(2X,'CURFIT: AA(',I3,') = ',IF15.5) DIAGN.EQ.8) WRITE(7,112) I,AA(I) J, SMATRX (I, J)  $\hat{D} = XX(N)^{**}(I)$ TO 22 CALL SOLVE(SMATRX, ICOEF) 0 0 J=1 ICOEF 99 I=1 ICOEF SMATRX(J,I) = (Itil CONTINUE 900 J=1 CONTINUE AUBSUM MUSUNA IF(IDIAGN DO 906 Q CONTINUE CONTINU AA AA . Э 4. 8 g g 8 800 8000 222 C 110 C 110 98 66 600 0000 υU

SOLVE A SET OF "NR" SIMULTANEOUS EQUATIONS USING GAUSSIAN ELIMINATION. FORMAT(2X,'CURFIT: AA(',I3,') = ',1F15.5) RETURN END 2,303,322 C(J,K)-C(J,I)*C(I,K) -1.D-5) 308,308,307 |C,J)/C(I,I) DO 902 I=1,ICOEF * ZNORM AA(I) = AA(I) = AA(I) * ZNORM IF(IDIAGN.EQ.9) WRITE(7,113) I,AA(I) CONTINUE DOUBLE PRECISION C(100,100), SAVE INTEGER NR  $\begin{array}{c} DO \quad 901 \\ AA(I) \\ AA(I) \\ CONTINUE \end{array} = , \underline{AA(I)} / \underline{XNORM^{**}(I-1)} \end{array}$ SUBROUTINE SOLVE(C,NR) L = I + KEXCHIF(L-NR) 309, 309, 330 kÊxch M NC = NR + DO 303 I=1 CONTINUE RETURN **PURPOSE:** 902 C 113 200 000 3022 3022 3022 301 307 307 υ 000000 00000 ບບ

PURPOSE: FIND THE MAXIMUM VALUE IN AN ARRAY. PURPOSE: FIND THE MINIMUM VALUE IN AN ARRAY. WRITE(7,331) FORMAT(2X, EQUATIONS CANNOT BE SOLVED...') RETURN END XMAX = X(1) DO 340 I=1 NITEMS CONTINUE AMAX = XMAX) XMAX = X(I) CONTINUE AMAX = XMAX RETURN END REAL FUNCTION AMAX(X, NITEMS) REAL FUNCTION AMIN(X, NITEMS)  $\begin{array}{l} \mathbf{KEXCH} = \mathbf{KEXCH} + \\ \mathbf{GO} \ \mathbf{TO} \ \mathbf{306} \end{array}$ DIMENSION X(500) DIMENSION X(500) INTEGER NITEMS DO 311 N=I,NC SAVE = C(I,N) C(L,N) 309 330 330 330 340 311 C ບບບ 000000  $\mathbf{U}\mathbf{U}$ 000000 0000

INTEGER NITEMS 340 00

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