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THEODORE NICHOLAS Project Engineer

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NATHAN G. TÖPVER Chief, Metals Behavior Branch Metals and Ceramics Division

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FOREWORD

This report was prepared by the Research Applications Divison, Systems Research Laboratories, Inc., Dayton, OH, under Contract No. F33615-79-C-5025, "Mechanical Property Characterization and Modeling of Structural Materials". The contract was administered under the direction of the Air Force Materials Laboratory, Metals Behavior Branch (AFWAL/MLLN), by Dr. Theodore Nicholas, Project Manager. The research reported here was conducted by Jalees Ahmad and was performed during the period June 1979 to August 1979.

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SECTION I INTRODUCTION

Finite element method is now well established as one of the foremost numerical techniques for analyzing linear elastic crack problems. Among other advantages associated with the use of finite element method is the fact that complicated geometries and physical loading situations can be quite accurately modeled. Therefore, the method is useful in analyzing standard as well as nonstandard crack growth and fracture specimens which may be used in the laboratory.

A disadvantage of the method is that with even a slight change in the geometry of a given specimen, such as the crack length, a new finite element mesh is required. Since finite element mesh generation is usually a time consuming process, parametric studies involving any change in geometry are tedious. Using general purpose large finite element computer codes, such as NASTRAN or MARC, for such problems is an inefficient and expensive process. However, if special purpose finite element codes developed primarily for parametric studies of test specimens are used, the method can be employed economically and with improved efficiency.

The present report provides a user's guide for a special purpose finite element code developed primarily for two dimensional linear elastic analysis of test specimens. User's guides for some supporting computer programs, such as mesh generators for commonly used test specimens and for plotting a mesh, are also included. Precise instructions and a number of illustrative examples are provided to clarify the data preparation for executing the program.

It is expected that a user with only a basic knowledge of theory of elasticity and some acquaintance with FORTRAN language should be able to effectively employ the procedures included in this report.

SECTION II PROGRAM HIGHLIGHTS

1. ASSUMPTIONS

- 1) Material is a homogeneous, isotropic, and linearly elastic solid.
- 2) Conditions of plane stress or plane strain exist.
- 3) Compared to the dimensions of the solid, displacements are small, and assumptions of linear theory of elasticity are valid.

2. METHOD OF SOLUTION

The numerical technique used by the program is based on displacement finite element formulation.

3. ELEMENT GEOMETRY

The element used in the present program is an eight node isoparametric quadrilateral 1 shown in Fig. 1. Some special and degenerate forms of this element are shown in Fig. 2. Element nodes are always numbered anticlockwise starting at any <u>corner</u> node. The four corner nodes are number (I), (I + 1), (I + 2), and (I + 3). The first intermediate node (I + 4) is <u>always</u> between corner nodes (I) and (I + 1). Other intermediate nodes are then numbered anticlockwise also.

4. MESH GENERATION

Mesh generation can be best explained by considering a simple example: consider the cantilever beam shown in Fig. 3. For simplicity let us decide to use only three elements. Fig. 4 shows some possible discretizations.

The mesh of Fig. 4a is most appropriate of all the other meshes shown. As a rule of thumb if the geometry of the problem so allows, curved, irregular and degenerate shapes should be avoided.

The node numbering should be such that the difference between any two node numbers belonging to the same element is kept as small as possible. This difference in Fig. 4a is 7 (=8 - 1). The above practice helps in reducing computational time.

Elements may be numbered in any arbitrary way. However it is a good practice to be systematic as in Fig. 4a. Size of each element, and the total number of elements in the mesh are decided depending upon the nature of a given problem. In general a finer mesh would provide more accurate results than a coarser mesh.

Mesh generation can be accomplished either by actually drawing the elements on a graph paper or by writing a mesh generation program. Usage of mesh generation programs for some specimen geometries is described in Appendix I.

5. TOPOLOGY

Complete topology of the mesh is described by individual element connectivities and X and Y coordinates of the nodal points with respect to any conveniently chosen X, Y coordinate system, as shown in the following example.

For the mesh shown in Fig. 4a the topology can be described as follows.

NODE	<u>x</u>	<u>Y</u>	NODE	<u> </u>	<u>Y</u>	NODE	<u>x</u>	<u>Y</u>
1	4.0	0.5	7	3.50	1.0	13	1.50	1.5
2	4.0	1.0	8	3.50	1.5	14	1.25	0.5
3	4.0	1.5	9	2.50	0.5	15	1.25	1.5
4	3.75	0.5	10	2.50	1.5	16	1.00	0.5
5	3.75	1.5	11	1.50	0.5	17	1.00	1.0
6	3.50	0.5	12	1.50	1.0	18	1.00	1.5

a. Nodal Coordinates

b. Connectivities

Following the numbering convention described in 3, the element connectivites are as follows.

ELEM	Ī	<u>I+1</u>	<u>1+2</u>	<u>I+3</u>	<u>I+4</u>	<u>1+5</u>	<u>1+6</u>	<u>I+7</u>
1	1	3	8	6	2	5	7	4
2	8	13	11	6	10	12	9	7
3	16	11	13	18	14	12	15	17

Note that I can be chosen to be any corner node.

6. PLOTTING THE MESH

After generating a mesh and punching out the nodal coordinates and connectivities on data cards it is often desirable to check for any topographical errors. Perhaps the easiest way is to plot the final mesh. Usage of a mesh plotting computer program developed for this purpose is described in Appendix II.

7. SPECIFYING NODAL DISPLACEMENT AND FORCE BOUNDARY CONDITION

For simplicity the program (SRLOIA) is designed to accept only zero displacement and non-zero force boundary conditions.

For the beam of Fig. 3 and using the mesh of Fig. 4a, we can specify zero X and Y displacement components at nodes 1, 2, and 3. Force of negative P can be specified in Y direction at node 18. The procedure for preparing boundary condition data is described in Section III.

8. SPECIFYING NON-ZERO STRESS AND PRESSURE ON ELEMENT SIDES

In problems where non-zero stress or pressure is to be specified (such as in crack problems with crack face pressure distribution), the pressure or

stress distribution over any side of the element is given in polynomial form. Constant, linear, quadratic, and cubic distributions are permitted. The method for specifying these distributions is discussed in Section III.

9. ENFORCING CRACK TIP STRESS SINGULARITY

The elastic stress singularity at the crack tip is included by surrounding the crack tip by elements of the form shown in Fig. 2e. The collapsed nodes (I + 1), (I + 2), and (I + 5) are placed at the crack tip and the <u>relative</u> displacements among these nodes are forced to be zero. This is accomplished either by specifying the X and Y displacements of the above three nodes to be zero, or simply by assigning a single number to all three nodes in the nodal numbering of the mesh. Details of this method can be found in the paper by Barsoum.² The use of the above procedure is illustrated in Section IV.

SECTION III DATA PREPARATION

1. USER'S GUIDE FOR SRLO1A

In this section the method for inputting topology, material properties, and boundary conditions data for the solution of a given problem is described. Each data set refers to a specific information required by SRL01A for execution.

Columns	Variable	Definition
1-4	NPRTYP	Problem Type (1 or 2).
5-8	NPOINT	Total number of nodes in the mesh.
9-12	NELEM	Total number of elements in the mesh.
13-16	NBOUN	Total number of nodes with displacement boundary condition.
17-20	NCONC	Total number of nodes with concentrated forces.
21-24	NFREE	Degrees of freedom per node (always give 2).
25-28	NYM	Number of different materials in the same problem.
29-32	NBAND	Expected band width.
33-36	IR	Number of nodes per element (always give 8).
37-40	NSTRSS	Stress computation indicator (0 or 1).
41-44	NCPOIN	Total number of corner nodes,
45-48	NQPTS	Total number of re-specified nodes.

a. Data Set 1 (sizing card), Format (1214), Number of cards = 1

Instructions

NPRTYP	Give	1	for	Plane	Stress	problem.
	Give	2	for	Plane	Strain	problem.

NPOINT Count and specify total number of nodes in the mesh (corner nodes + intermediate nodes).

NELEM Count and specify total number of elements in the mesh.

- NBOUN Count and specify total number of nodes which are fixed in either one or both X and Y directions.
- NCONC Count and specify total number of nodes which have concentrated applied forces in either one or both X and Y directions.
- NYM In general each element in the mesh can be allowed to possess different elastic properties. If so, specify the total number of materials to be used. For uniform elastic properties for all elements give NYM equal to 1.
- NBAND This is a sizing parameter found by the following formula. NBAND = (Max. difference between any two nodes of an element +1) × 2.
- NSTRSS For crack problems where stress intensity factors are of interest give NSTRSS = 0. For other problems where element stress and strain components are required give NSTRSS = 1.
- NCPOIN Total number of corner nodes (NPOINT number of intermediate nodes) in the mesh.
- NQPTS If NQPTS is given zero, the intermediate nodes are automatically placed at the middle of straight line distance between the adjoining corner nodes. For problems in which this arrangement needs to be altered, such as in crack problems using the element of Fig. 2e, the number of those intermediate nodes which have to be moved from the mid-position should be counted and specified as NQPTS.

b. <u>Data Set 2 (Elastic Property Cards)</u>, Format (2F20.5), Number of Cards = NYM^{*}

Columns	Variable	Definition
1-20	El	Young's Modulus
21-40	P1	Pois <mark>son's Ratio</mark>

*See a. Section III.

c.	Data Set 3	(Boundary	Conditions),	Format	(414),	Number	of	Cards = NBOUN

Columns	Variable	Definition				
1-4	NF(1)	Node with displacement boundary condition				
5-8	NB(1,1)	Zero X-displacement indicator				
9-12	NB(1,2)	Zero Y-displacement indicator				
ļ	1					

Instructions

- NF(I) Give the number of a node whose X or Y or both displacements are fixed.
- NB(I,1) Give 1 if X-displacement of node NF(I) is fixed. Give 0 if X-displacement of node NF(I) is not fixed.
- NB(I,2) Same as NB(I,1) but for Y-displacement of node NF(I).

* See a. Section III.

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d. <u>Data Set 4 (Concentrated Loads)</u>, Format (14,2F20.10), Number of Cards = NCONC^{*}

Columns	Variable	Definition
1-4	NP	Node number of point with conc. load
5-24	U (NP×2-1)	Value of X-component of applied load
25-44	U (NP×2)	Value of Y-component of applied load

* See a. Section III.

e. Data Set 5 (Corner Point Coordinates), Format (14,2F10.5), Number of Cards = NCPOIN*

Columns	Variable	Definitions
1-4	I	Corner node number
5-14	X(I)	X-coordinate of I
15-24	Y(I)	Y-coordinate of I
1		

*See a. Section III.

** For example in Fig. 4a points 1, 3, 6, 8, 11, 13, 16, and 18 are corner nodes (NCPOIN = 8).

f. Data Set 6 (Element Connectivities), Format (914, F10.5, 14), Number of Cards = NELEM*

Columns	Variable	Definitions
1-4	I	** First corner node
5-8	I+1	Second corner node
9–12	I+2	Third corner node
13-16	I+3	Fourth corner node
17-20	I+4	First intermediate node
21-24	I+5	Second intermediate node
25-28	I+6	Third intermediate node
29-32	I+7	Fourth intermediate node
33-36	NEP	Material property number
37-46	THICK	Element Thickness
47-50	NTYPEL	Applied pressure/stress indicator (0 or 1)

Instructions

- NEP If a single material is used (NYM = 1 in Data Set 1), NEP is 1 for all elements.
- NTYPEL If any side of an element (see Fig. 1) is subjected to applied pressure, for that element give NTYPEL as 1. For other elements NTYPEL is given 0.
- * See a. Section III.
- ** Follow the numbering convention of Part 3. (Section II)
- g. Data Set 7 (Crack Tip Node), Format (14), Number of Cards = 1

Columns	Variable	Definition			
1-4	NTIP	Node number of a node located at the crack tip *			

*If there is no crack give zero.

h. Data Set 8 (Coordinate Modification), Format (I4,2F10.5), Number of Cards = NQPTS^{*}

Columns	Variable	Definition
1-4	I	Node number of the point whose coordinates are to be modified
5-14	X(I)	X - coordinate of I
15-24	¥(I)	Y - coordinate of I

See a. Section III.

and the second of

i. Data Set 9 (Pressure Polynomial), Format (I4), Number of Cards = M Format (8F10.5), Number of Cards = M

Card 1

Columns	Variable	Definition Side number of element on which pressure			
1-4	NSIDE	Side number of element on which pressure is applied (See Fig. 1)			

Card 2

Columns	Variable	Definition
1-10 11-20 21-30 31-40 41-50 51-60	Variable A ₁ A ₂ A ₃ A ₄ B ₁ B ₂	First coefficient of T _y traction polynomial Second coefficient of T _y traction polynomial Third coefficient of T _y traction polynomial Fourth coefficient of T _y traction polynomial First coefficient of T _x traction polynomial Second coefficient of T _y traction polynomial
61-70 71-80	в ₃ В4	Third coefficient of T_x traction polynomial Fourth coefficient of T_x traction polynomial

This is an optional data set. Use only if one or more elements have NTYPEL = 1 (See f., Section III). If there are no such elements omit this data set altogether.

*M is the number of elements with NTYPEL = 1. For each such element there are two cards. The set of Card 1 and Card 2 has to be given for each such element. Give M such sets. The order of the sets is the same as the order of element cards in Data Set 6 which have NTYPEL = 1.

(Continued at top of next page)

 T_x and T_y are the x- and y- traction components respectively, obtained by taking the vector dot product between the stress components and the outward unit normal on the surface.

j. Data Set 10 (KI Computation Indicator), Format (14), Number of Cards = 1*

Column	Number
4	1

*If NTIP=0 (see g. Section III), this card is omitted.

k. Data Set 11 (KI Computation Point), Format (14), Number of Cards = 1*

Columns	Variable	Definition
1-4	NKIC	Nodal point on the crack edge whose displacement is to be used for K _I calculation. NKIC should be approximately A/50.0 distance away from NTIP, where A denotes the crack length

^{*}If NTIP=0)(see g. Section III), this card is omitted.

2. LIMITATIONS

For using the stored version of SRLO1A the following imitations on the values of various variables should be observed.

 $0 < NPOINT \le 340^*$ $0 < NELEM \le 100$ $0 < NBOUN \le 50$ $0 < NBAND \le 100$ $0 < NYM \le 5$

For definitions of these variables see a. Section III.

SECTION IV

SOLUTION PROCEDURE

1. PROGRAM EXECUTION

The necessary control cards for executing the program with a single data set (one problem) or multiple data sets (many problems) are as follows.

First Card (Job Card), start at column 1.

AAA, CM300000, T100, I0100, STANY. M760328

AAA = Job identification number (eg. user's initials). T100 = Execution time limit in seconds. This may be changed (eg. T60) I0100 = Input-Output time limit in seconds. This may be changed. M760328 = Example of user's code. Use appropriate account number.

Note: Put a period, then a blank after STANY.

Second Card (attach card), start at column 1.

ATTACH, LGO, SRLO1A, ID=M760328.

SRL01A = Program name. For other programs described in appendices the program name is changed but other parameters remain identical. For executing SRL01A finite element program use the above card exactly the way it is shown.

Note the period after the ID value.

Third Card (load-go card), start at column 1.

LGO,

Fourth Card (end-of-record), this card is available at the computer center.

CARD SEQUENCE

The necessary control cards and data cards have to be arranged in the following sequence.

Job card

LGO.

Attach Card

As many as number of problems to be solved.

LGO.

End of record card Data for first problem End of record card Data for second problem

End of record card Data for the last problem End of job card.

SECTION V

NUMERICAL EXAMPLES

The purpose of providing solved numerical examples using the programs discussed in the present manual is to further clarify the data preparation and program execution described in previous sections.

1. EXAMPLE 1: SINGLE NOTCH SEMI-CIRCULAR SPECIMEN

In order to explain all the procedural details, let us start with the problem of a semi-circular ring containing a radial crack shown in Fig. 6. Due to axial symmetry about the X-axis only one half of the geometry is needed for analysis (Fig. 6b). To account for the other half, we need to specify Y-displacement (V) to be zero on the boundary indicated in Fig. 6b.

The purpose is to obtain nodal displacements and stress intensity factors (K_I) for various values of crack length (A), inner radius (R_i) , and outer radius (R_0) . We proceed in the following steps:

a. Step 1: Mesh Generation

As explained in Section 2, mesh generation can be accomplished either by hand or by using the computer. Let us use SRLC program of Appendix I to generate the mesh. The two data cards used were:

Card 1: Blank Card 2: 0.5 1.0 2.0

The numbers in card 2 represent A, R₁, R₀, respectively.

The output of SRLC in the form of punched deck, and in print, was as follows:

	50003 1.	66000	2.0	0000				
1	185 52 1	17 1	2	1	70	8 C	57	5
	10000000.	00000				.30000		
105	<u>61</u>							
166	し 1							
107	0 1							
108	ü 1							
169	0 1							
114	0 1							
123	ü 1							
128	0 1							
137	0 1							
142	0 1							
151	u 1							
156	u 1							
165	0 1							
170	0 1							
179	<u> </u>							
184	U 1							
185	1 1							
13					1.		100	
1	• ŭ j U i ŭ	1.000	80					
3		1.112	56					
5	• ម៉ុមម៉ូមិម៉ូម	1.225	UU.					
7	• 00000	1.500	ΟU					
9	• (00000	1.775	ថដ					
11	• 6 6 6 6 9 9	1.887	50					
13	• ដីមីមីមីមី	2.000	00					
21	38268	• 923	88					
23	42574	1.027	82					
25	46679	1.131	75					
27	57413	1.385	82					
29	07920	1.639	89					
31	72231	1.743	82					
33	76537	1.847	76					
41	70711	.707	11					
43	70606	.786	uб					
45	 00621	.866	21					
47	-1.0óló6	1.060	66					
49	-1.25511	1.255	11					
51	-1.33400	1.334	66					
53	-1.41421	1.414	21					
61	92308	• 3ö2	68					
63	-1.02782	• 425	74					
65	-1,13175	. 458	79					

į.

67	-1.38582	.57403						
69	-1.53989	.67926						
71	-1.74382	.72231						
73	-1.84776	.76537						
81	990.00	.11006						
83	-1-11250	-11000						
85	-1.22500	.11000						
87	-1.50000	-11000						
89	-1.77500	-11808						
04								
74	-1.00/90	-11004						
104	-1 000.00	•11000						
102	-1 1100000							
406	-1.226.00	0.00000						
104	-4 22500	8.00000						
4.07								
107	-1.85/90	0.00000						
103		0.00000						
115	-1.33/90	0.10000						
117	-1.33/20	• 0 9 6 6 8						
119	-1.90600	• 19110						
121	-1.00290	.09000						
123	-1.55290							
1 24	-1.39379							
1 2 2	-1-34379	. 47560						
176		. 67500						
127	-1.536.5	• 0/500						
11.7	-1.01022							
143	-1047644							
47	-1 - 6 - 6	• 05000						
140	-1.5.0000 -1.5.0000	• USU UU						
121		. 05000						
167	-1.599999							
150								
161	-1.30000	• U 2 U U U						
163	-1.226.30	• U Z U U U						
155	-1.52000	02000						
1/1	-1.63							
173	-1.56203	0000000						
175	-1.5.6.04							
177	-1.3.7.7							
179								
185	-1.5.4.4	0000000 3.36000						
	23 21 1	15 22	4 1.	2	.,		10.500	
5	25 23 3	16 24	4.5	<u>د</u>	4	1.	មាមាមាមីមី កាលការក	
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9	29 27 2	18 28	12	5 2	4	. 1 .	10161	
11	31 25 9	19 30	1 4	1.6		4.	មមមមម្រំ លោកតេ	
13	33 31 11	20 32	19	12	1	• 1 ·	30305	•

23	4 3	•1	21	35	42	34	22	1	1.40040
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23		47	27	3.4	LA	37	28	1	1.05066
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22		- 	31		50	70	30		1.4.300
		21			50				1 1 1 2 0 0
43	03	- 01 	44	27	02	24	46	- <u>+</u>	Te 10 0 0 0
42	02	03	43	20	54	22	44	4	TEATION
47	01	02	42	27	00	20	40	T	
49	03	37	47	20	00	21	40	1	1.0000
21	1 1	27	49	23	10	50	50	1	1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1
23	13	/1	51	50	(2	29	52	1	1.0.000
63	83	81	61	75	82	74	62	1	1.30000
65	62	33	63	76	84	75	64	1	1.]uÜüÜ
67	87	65	65	77	85	76	ъб	1	1.33366
69	89	07	67	78	88	77	68	1	1.43360
71	31	ъэ	65	73	Ϋ́υ	78	7 ü	1	1.3.33
73	73	37	71	80	92	79	12	1	1.00000
81	83	102	166	82	92	1 1 1	94	1	1.00930
83	85	104	102	84	96	103	95	1	1. JÜ]ÜĹ
83	91	107	165	90	98	105	97	1	1.0.000
91	93	109	107	92	93	148	98	1	1.33363
85	117	115	104	111	115	110	96	1	7.90999
85	87	119	111	86	116	115	111	1	1.00000
87	89	141	119	88	113	126	112	1	1.03000
121	69	103	123	113	97	114	122	1	1.00000
117	131	129	115	125	13ú	124	116	1	1
117	119	133	1 31	118	126	132	125	1	1.44300
119	121	130	1 33	128	127	134	126	1	1.30800
135	121	123	1 37	127	122	128	136	1	1.0.0003
131	145	143	129	139	144	138	1.30	1	1.00000
1 31	133	147	145	132	144	146	139	1	1.000000
133	135	149	167	134	141	168	146	1	1.3.3
149	135	1.37	151	161	135	142	160	1	1
145	1-59	1.7	143	153	154	1 5 2	144	1	1.0.160
147	149	103	161	146	165	162	154	1	1.00343
145	147	151	164	146	166	164	453	1	1.03358
163	160	1 - 1	165	155	160	1 5 5	164	1	1.0.300
164	173	171	167	167	172	166	158	1	1.00300
150	161	174	172	160	1	174	167	4	1_80300
461	157	127	17-	462	16.0	476	1.5.9	-	4 12300
477	103	1-5	170	102	107	170	100	1	4 35360
472	400	107	473	107	4 6 5	1 0 0	470	1	T.0.0.000
170	107	107	477	101	107	TOU	112	1	
1/7	107	107	173	102	107	101	1/4	1	
1//	105	105	1/5	103	102	182	175	1	1.00000
1/9	182	102	177	184	182	183	178	1	1.00000
185	-			. .					
100	-1	4971	20	U . U	1000				
181	-1	438	23	• 0 (1177				
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183	-1.	5117	(7	• Ü l	1177				
184	-1	5125	うし	VeJi	1006				
1									

and the second secon

The first line in the output gives the crack length and the inner and outer radii of the specimen. The rest of the computer print-out corresponds to the data sets described in Section III as follows.

Line 2	Data	set	1
Line 3	Data	set	2
Lines 4 to 20	Data	set	3
Line 21	Data	set	4
Lines 22 to 88	Data	set	5
Lines 89 to 140	Data	set	6
Line 141	Data	set	7
Lines 142 to 146	Data	set	8
Line 147	Data	set	10
Line 148	Data	set	11

Note that data set 9 is missing because no element side is subjected to applied pressure.

b. Step 2: Mesh Plotting

From the punched data deck of step 1, data sets 5 and 6 (coordinates and connectivities) were temporarily removed for use in the plotting program SRL11. In accordance with the description provided in Appendix II, the following data was supplied to SRL11.

6 185 52 [Nodal coordinates of data set 5 above] [Blank card] [Connectivities of data set 6 above] 1 20 1 C-SPEC 0.0 2.0 -2.0 -1.0 3.0 3.0 1.0 1.0 [Blank card] End-of-record.

By executing SRL11, the plot shown in Fig. 7 was obtained. In Fig. 8 details of the fine mesh near the crack-tip are shown. Fig. 8 was obtained by supplying SRL11 with only those element connectivities which were to be plotted and by increasing the scales (XSCALE, YSCALE). The number of elements (52) in the first data card corresponds to the number of elements which are to be plotted.

c. Step 3: Executing SRL01A

Data sets 5 and 6 were placed back in their proper sequence in the punched data deck of Step 1 after removing the blank card which was inserted behind data set 5 for plotting purpose. SRLOLA was then executed by using the following cards.

Job card ATTACH,LGO,SRLO1A,ID=M760328. LGO. End-of-record [Data deck of Step 1] End-of-job.

The output of SRLO1A was as follows.

		X-C00RD	Y-CUORD	X-DISPL	Y-DISPL
NODE	-1	0 • v000 d	1. F03UC	55443382-00	• 11185945-04
NODE	N	0.00000	1. C 5625	5948113E-05	.11187065-04
NODE	m	0. 60566	1.11 23u	5349219E-05	•1118936E-n4
NODE	4	C. Cu000	1.16875	o749467Ľ-65	• 1113253E-04
NODE	الل ا	0.600.0	1.22510	7147ú88E-05	• 1119793E-04
NODE	۱D (0-0-0-0	1. 3625C	8119217E-05	 11231205-34
PCON		ü • va009	1.55000	3034080ビー15	.11278922-34
NODE	ø	U + U O Ü O	1.63750	1r1926nc-64	. 113676fc-04
NOOL	σ	0.60453	1.775uu	11129655-04	. 11463665-34
NODE	10	<u>0.60000</u>	1.83125	11959696-04	• 1151 695E-04
NODE	11	9.0000	1.88750	1232275E-U4	• 11559845-54
NODE	12	U. UOOÚÚ	1.94375	12497 37E-04	.1162922E-U4
NODE	13	L.C0064	2.56940	1239287E-C-	. 11387310-04
NODE	4 4	19134	• 96194	5279507 <u>-</u> 05	- 97948655 •
NODE	15	21287	1.57016	6053292F-05	. 36384425-05
NODE	16	23440	1.17838	- - 58171255_05	 J48J123E-05
NODE	17	28702	1.44291	8657729E-U5	.9141391 ^c -05
NODE	18	33963	1.76745	1057126 <u>6</u> -04	• 8777334E-05
NODE	19	36116	1.81560	1138719L-64	. 801453fi-05
NODE	20	33269	1. 52348	1222r93E-u4	• 9444207 <u>2</u> -05
NODE	21	38258	.92388	5(32954E-U5	• 8367+73E-US
NODE	N V	43421	.97585	- . 54	• 32067682-0F
NCOL	23	42574	1.62782	5772529L-05	. 3U5u 8155-05
NOOF	5	44727	1.1979	51353542-05	. 78374515-05
NUDE	52	46879	1.13175	6446905-05	• 7746138E-05
NODE	0 V	52141	1.25379	7371502E-05	• 7 3 9 1 3 5 2 E + 9 E
NODE	27	57403	1.38582	82315645-C5	.7015532-15
NODE	28	62065	1.51205	31442956-93	• 66533245-8F
ACON	29	č7926	1.63989	10189655-64	 52842265~05
NOOE	- M	73479	1. e 9186	-• 1c +22 45E-84	• 6139021E-05
NCOL	31	72231	1.74332	1r795b304	• 5 ¥ 31 7 7 15-45
NOUL	32	10012-	1.79579	1115¢0 2∟-04	• 2 R 5 3 G 9 5 1 - 0 5
NUDE	<u>ی</u> ۲	70537	1.84776	- .11 328355-64	• 5715u1505

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50-7 • 5275675=35 • 5276399E=05 424195/E-05 3012551E-05 23395396-05 • 2145118E-05 • 1830 455E-05 .13775595-05 56758512-05 2509491E-05 5935101E-06 39787615-05 . 3657129Ľ-05 33551205-05 22770465-35 883943 fE-06 71434055-35 **3098147 -15** +2867335-05 . 3889 869 80 9E - US 34681485-35 59172156-05 5339267E-05 j119021E-05 .48613962-05 36258945-15 ŝ 4512887E-05 39355862-05 1239784E-05 1+3+3565-06 42826562-05 3042143E-US .15582492-35 .1634A205-09 <u>5644835.</u> -. 56153 152-05 -. 725347 56-05 -. 751781 36-05 -. 5598607E-05 -. 7114505E-05 53584642-05 -- 5985071E-C5 -. 3164694c-95 -. 3527837E-05 -- 4419212E-05 --533919E-05 --5112213E-05 -. 195322 aE-05 -. 2311172E-65 -.2414869E-05 -.26332456-05 -.2964792E-05 -. 39754 452-05 -- 42295 J. 9E-05 -. 4482348E-05 -. 47391 DAE-05 -. /782252E-05 --80487395-05 -- 93156 n 3E-05 -. 2816917c-05 -. 5720674E-05 -. 2055293E-u5 -. 2193547E-05 -. 32389996-05 -- 4978685L-05 -- 8629314E-P5 -- 926227 FE-US --96387716-05 -. 37217652-05 -. 4347320E-05 ľ . 56719 .74089 1.29439 1.33466 1.02349 •42574 46679 .57473 81551 +2724 8606F. .22324 1.44750 1.63699 .7(711 . 78506 • 5 2 6 4 4 •96344 • ـ ى 2 ق ف 1.15789 1.25511 1-37444 ·5++3· . 6000 .66750 .81735 1.58979 .35268 v2665 1.53324 . 66621 1.41421 .40421 .44727 52141 -1.23379 -1.51236 --14492 -- 60750 -- 41735 -.96719 -1.2849 -1.(8979 -.74683 -.78000 --82044 -- 96344 -1.15789 -1-24485 -1-33400 +++2001--1.41421 -.81550 +2169 --- 44858 -1.22324 -1.44750 -1-53924 -1.63799 -. 52338 -- 47585 -1. J2732 -1..7979 -1.13175 -1.30582 -.74711 -- 62621 -1.25511 30 30 50 80 36 6 M ₹ E 5 10 1 ST T N M ŝ 80 53 ± ± e. E د ۵ 62 t c Ω Ω 0.0 5 3 د. ح 끜 ور افر 2 500 3 57 ŝ 5 NUDÉ NODE NODE NODE NCDE 300N NUUL NCDE NOUL NODE NODE NODE NODL NCDE **NGD** A ODL NOJE N.J.C. NCDE **NCOL** NODE NODE NGUL NCOL NODE NODE N ODL **NUDL** NCOL **NODL** NODE NCDE NODE NUDE

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-00 -E-36 50-3 -. 4735020E-06 3813443E-05 30960215-05 24193346-05 .1435555-05 0183635-07 -- 1070264E-96 -- 3446869E-36 -. 6357278E-16 -- 3394119E-06 .4623225-05 • 3370069E-u5 5 11114592-05 -- 8891836⁻-06 .34627315-05 27566982-05 -- 1629965E-06 -- 21796355-06 -. 2551877E-06 -- 2957478E-06 37857385-05 30981712-05 24073855-05 -- 1371 3925-06 • 3759396E-05 .30994712-05 2124735E-06 62371395-97 -, 6531651E-07 -• 5776023E-97 275J5utE-05 2699293E-0! 266 0161 -. 1129885 + 3430741 ŝ • . . ē • • • . • . -.3781216£-05 -• 3917469E-05 -. 1243069E-U5 --1359335E-05 -.1492764c-05 -. 17421 u 7E-05 -- 1048544E-05 -. 1995¤61E-05 -. 213531 8E-05 -. 49750226-06 -. 4935758E-06 -. 5058973E-05 -.31588162-06 -.53224825-06 -- 6177528E-u6 -• 6631824E-06 -. 581.77072-06 -. 74850312-06 -- 522434 AL-06 --8503495E-06 --8837524E-06 -- 18507 195-06 -.1885385E-05 --20384451-06 --65908147-05 -- 7525147L-00 - 83458135-05 13509832-66 3523913E-05 -.3643872E-05 -- 4059693E-05 -. 78353215-06 13397 09E-05 13580625-06 13544075-06 . • . 6787 .43769 70079 74384 41610 908TT. 11900 . 45565 . 15540 • L 55 56 • • • • • • • 72231 6537 24634 2894C 34262 39463 .11UUC 10011 .11006 11000 .11006 110 . (.11050 11000 11000 110 uL .11006 10451. ا.دناگ.٩ U- 6 6 8 6 6 6 2.6C35u 67926 (••150) ŝ -1. J125 1.63989 -1.69186 -1.74382 -1.79579 -1.84776 -- 95694 -1.07616 -1.17830 -1.76745 -1.81566 -1-91388 -1.11250 -1.16875 -1.22500 -1.50006 -1.63750 -1.08750 -1.93375 -1.44291 10065 *--1.36258 -1.77550 -1.63125 -1-53549 -- 995u -1.1125 U -1.225 v û -1.77530 39645 -1.45625 ---11255 -1.83755 -1.5.5.50 -1.16375 100+NM+000 52 82 84 85 ðb 87 88 59 6 455 63 95 96 56 liu 101 103 5 102 51 35 96 NODE YUDE NOOL KOOE NODE KUDE NOUL NOOE ROUE NOUE NODE NODE NODL NODE NODE LUDE NODE NUDE NOUL 600E 200F NODL **VODE** NODE KODE CODE NOUL 1005 NUDE VOOL **LUUL** KODL AUUE KOUL

•	-1.22500	0 - 6 - 5 - 5	• 1346449C-66	• 24968182-ù5																														
165	-1.77500	30000.00	624359AE-A5	• 7																														
146	-1.83125	0.0000	F890813E-06																															
167	-1.88750	C • c 44 c •	7323245E-05	J.																														
108	-1.94375	6 - 7 6 c - 0	- -78133232-06																															
1 ú 9	-2.60130	ŭ - 5 ŭ ŭ G Ë	-• 82332 25E-06	ů.																														
116	-1.28125	0.1001.	 12525J3z-00 	.20370222-05																														
111	-1.28125	60377.	5(84189E-06	· 26229182-95																														
112	-1.56336	.12036	6222175E-05	• 47466546-36																														
113	-1.71875	.16036	5510433c-06	54831725-67																														
114	-1.71875	1.1000	5635618E-UG	0.																														
115	-1.33750	100-0-1	 10234295-05 	• 16+87276-35																														
110	-1.33754	• i 458C	20153 <u>04</u> 5-46	• 16414JJE-0F																														
117	-1-33756	0 ŭ ŭ Ū	48645 J25-05	. 16319568-85																														
118	-1.41875	- č 93 u č	- 5 417370E-06	.16518335-35																														
119	-1.51000	.09000	58356914-06	.44396212-76																														
129	-1.58125	1061.	5633249E-06	 16138355-06 																														
121	-1.06250	.09003	5260260 E -05	51453432-08																														
122	-1.66259	• 0 4 0 0 0	5234682 <u>2</u> -06	<i>5</i> 4171802-38																														
123	-1.00250	5.66005	4857085 <u>0</u> -06	. J.																														
124	-1.30563	G • C (^))	.8768525E-u7	 14916068-05 																														
125	-1.36563	• L 8250	45338212-06	 1435194≦-05 																														
126	-1.56036	• 6 8250	5514209E-05	• 4335291E-06																														
127	-1.63438	• U 825U	-• 5817849E-05	.35197552-07																														
128	-1.63438	0.66406	4 b 23C 7 2E - 66	.																														
129	-1.33375	0.0000	 70149056-07 	.124939E5-n5																														
130	-1.39375	• ü 3750	1970C43E-0b	• 1239674g-05																														
131	-1.39375	.07590	43774462-36	• 1232279 <u>5</u> -05																														
132	-1.44638	. u750C	46 3 7 5 6 4 5 - 4 5	. 83202257-06																														
433	-1.50900	.37536	5186338£-06	• 4101734F-06																														
134	-1.5313	• 675 86	5153561E-L6	• 2033534£-66																														
135	-1. čJ 625	.07500	53741 97E-Co	.755531157																														
136	-1.60025	• 53755	-• ¢339444105	• 3214798L-h7																														
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138	-1.42188	0.0000	• 3275899 <u>6</u> -07	• 1 [363432-05																														
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NODE 139	NODE 140	NODE 141	NODE 142	NODE 143	NODE 144	NODE 145	NODL 146	NODE 147	NODE 148	NODE 149	NCDE 150	NODE 151	NODE 152	NUDE 153	NODE 154	NODL 155	NODE 156	NUDE 157	NODE 158	NODE 159	NODE 16C	NODE 161	NODE 162	NODE 163	NODE 164	NOUE 165	NODE 166	NODL 167	NODE 168	NODE 169	NODE 176	NODE 171	NODE 172	NODE 173

NUDE 174	-1.43647	• 6 i a 9 u	1653734-65	. 218767556
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P.CDE 177	-1.51767	. UV7 47	14 35 P A 4 - 4 10	• 3651158c-u7
NGJL 178	-1.23854		12394J5L-06	• 2333771E-07
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NOJE 160	-2.4 3755	いでしょう・フ	· 1079437L-L9	.17399965-06
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NOJL 102	-1.50.JU	042y1 *	8[92338L-07	. 912237(E-67
NUJE 103	-1. pul77	-L-1/7	7642881E-u7	• 289459257
NUDE 164	-1.50250	0.0000	6011956E-07	a.
NCOE 165	-1-1-10	0 3 (i - i - i - i	•	0.
THE CRACK	TIP IS AT NUDE	185		
		1 991 ILS ###	ALENSITY FACTORS ***	
USING THE	Y-UISPL OF HUL.	. 171 Kit	21.J8659	

d. Step 4: Interpretation of Results

The K_I value for the problem is computed by the program and is given at the end of the output. Displacement values at various locations, such as crack mouth opening displacement (node 100) and load point displacement (node 13) can be read directly by identifying the proper node number on the mesh. It should be noted that the actual crack mouth opening displacement and total load line displacement in the present case will be twice the Y-displacement of nodes 100 and 13, respectively. The results are:

 $K_I = 21.58609 \text{ psi}\sqrt{\text{in}}$, Load Point Displacement = $233.74620 \times 10^{-7} \text{ in}$. Crack Mouth Opening Displacement = $75.1878 \times 10^{-7} \text{ in}$.

e. <u>Remarks</u>

AAA, CM7000, T100, I0100, STANY. M-----ATTACH, TAPES, SRLC1, ID=M760328. ATTACH, LGO, SRLC, ID=M760328. LGO. End-of-record 3 (Total number of cases minus 1) 0.2 1.0 2.0 0.3 1.0 2.0 0.4 1.0 4.0 0.5 1.0 4.0 End-of-job.

The above set-up will produce data for crack lengths 0.2, 0.3, 0.4, and 0.5 for $R_i = 1.0$ and $R_0 = 2.0$. The job set-up for multiple runs using SRLO1A is described in Section IV.

Note that the last four elements are as shown in Fig. 2e. Nodes 180, 181, 182, 183, and 184 are located at quarter points of the element sides. This necessitates the specification of the coordinates of these points in data set 8 (Section III). The purpose of this particular procedure is to model the proper stress singularity or the crack tip as mentioned in Section II.

2. EXAMPLE 2: DOUBLE NOTCH RING TENSION SPECIMEN

Consider a double notch circular ring (Fig. 9) under diametral tension. Due to symmetry about X and Y axes only one quarter of the ring needs to be considered for analysis. In fact by changing only the boundary conditions the same mesh as of Example 1 (Fig. 7) can be used. This is accomplished simply by changing the data sets 1, 3, and 4 of Example 1. The following computer print-out shows the changes in the above data sets.

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Note that in data set 1 the number of boundary conditions (NBOUN) has been changed from 17 to 30. In data set 3, thirteen more boundary conditions have been added and the boundary condition of the crack tip has been changed. In data set 4, the load is now applied at node 1. Also, since the applied load is 1.0 lbs, the resulting values for displacements and KI should correspond to a 2.0 lbs load on the actual ring.

The rest of the solution procedure remains identical to that of Example 1. The computer print-out of the results obtained by SRLO1A is shown on the following pages.

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1		X=C00X C	Y-000-Y	X-DISTL	Y-DISPL
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400k	44 41		1.88754	•	• 1845491 °-65
NODE	1.7		1.94375		• 18 35428 E-45
NODE	۲ ۲		2. 51.51	•	 1825764E-05
JUCN	14			22512371-56	19156.65-45
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ADDE	۲1 ۲4	- 53 76 8	•9×383	 3666 28 78-96 	. 17521535-45
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4005	r. 2	- • 42574	1.762	• 265477rr+66	• 178 4435 E- 15
NODE	24	++727	9767.1.1	. 21524475-16	• 15876465-85
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30Ch	-1-			.27:23145-06	. 157 2224 E- 65
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E 127	-1.60625		• 16: 1121E-05	•0
110	-1.42185		• 17(52°2E-15	• 26165J3E+16
E 1.70	ニスポンサ・ス・	• 525.	•1503967E-15	• 2793674 5-v6
	-1.5:00.	1.625	21-322275 - C 2	
E 1 4 1	-1.57 81 3	• F 5257	.1573496E-05	• 3626637E-57
F 142	-1.57 21 3	10.1.0	• 1613659E-05	
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44	-i.450L	•		. 2187327F-u6
5 145		1335 J.	• 16+1458F-F5	• 2185992 E-06
E 166	-1.47500	• 5:44	.15933675-65	. 1601.665E-06
E 147	-1.50000	• 0 50 to:	• 159° 987E-15	. 100 3463 5- 45
E 142	-1 - 52 500		• 159726 6F - 65	. 6316, 6. £-07
r 149	-1.555.1	1.5 1.5 1.5 1.5 1.5	• 159349 F + FF	. 42012866-67
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5152	-1.4655		.17.17545-05	· 1821,141 E-66
1 453	-1.405'	• 1 355 T	• 16277375-35	• 161 07845-UF
124	-1.50 1.6"	• • 35.1;	• 1611 1985-E5	• 842 P 0 3 5 - 6 7
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156	-1 - 57 51		• 164 2295-15	•
157	-1.4301	3.03856	• 17 11 75 3E - 15	• 1366938 E-0 f
158	-1.48761	• 5 2 3 Ŭ E	•16680705-05	. 13556815-06
159	-1-40070		• 1642675E-US	• 1355420 E-VE
160	3664.1-			• 115 27745-56
F 161	-1-53765		• 16 3428 9E-75	• 6439595E-47
162	-1.51.1		•1635064E+45	• 3979456E-#7
E 463	-1.52406	40 h 2 4 * *	• 16341785-25	• 27 2 1 2 4 3 E - 1 7
- 164	-1.52060		• 16472146-15	
11 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	-1.52300	000000 °C	• 165420 3F-f5	
F 165	-1.485: .	10.77	• 17 18 52 25-1 5	• 1176698 5-06
F 167	-1.48647	.61354	.1653435F-P5	• 1111 461 E-C6
169	-1.59660	• 1 22 [-	• 1 54378 GE-PS	• 5575581E- 1
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3UCA	17C	-1.5151-	<u>, fr: ar</u>		
	171	-1 - 49 7 (3:		• 17i 0 40 55-05	• 9596299 ^c -07
NOUT	1.72	-1.49147	• 0 03 54	• 164267 PF-65	• 6817671E-07
NUDE	173	-1.49293	• [- 7 7	• 166690 6E-85	• 300 6869E-C7
ULON.	174	-1.40.647	• <u>52654</u>	• 1 05917 · E · U 5	• 6143543E-f7
BUON	175	-1.5296	200 100 €1 €1	• 1654334E-05	. 45748125-07
30Ch	176	-1.5354	• : 3354	.1658779F-75	· 2823840E-67
30CN	177	-1.5-7-57	• . 7 . 7	• 1661J89E-75	.1524361 E-<7
NOPE	173	-1 + 52 554	• r 3 54	•166657 <u>2</u> E-25	• 6741 946 E-1, 8
JUCK	170	51 11 :	2.4 0 3 6 5	.16671495-n5	•
JUCK	1.9 1	-1.4375		. 17: 21 58-05	• 4778139E-07
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FOCK	1.8.7	-1+52177	22 772 *	• 168 549F-55	• 8185551 E-1.8
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As in Example 1, the K_I value corresponding to the applied load (2.0 lbs) is given directly. Nodes 1 and 100 correspond to the load point and the crack mouth, respectively. Accordingly, we have for 1 lb. load:

 $K_{I} = 3.0068 \text{ psi}\sqrt{\text{in}}$

load point displacement = $22 \cdot 34021 \times 10^{-7}$ in. crack mouth opening displacement = $9 \cdot 511695 \times 10^{-7}$ in.

3. EXAMPLE 3: DOUBLE NOTCH RING COMPRESSION SPECIMEN (CRACK LINE PRESSURE)

The geometry of a Double Notch Compression (DNC) Specimen is shown in Fig. 10a. Let us choose A = 0.5 in, $R_1 = 1.0$ in and $R_0 = 2.0$ in so that the mesh of examples 1 and 2 may again be utilized. For this case however, instead of obtaining a solution for a concentrated load directly, as in Example 2, let us use the crack line pressure concept; the purpose being to illustrate the procedure for applying distributed loads.

In Fig. 10b σ_{yy} is the crack line pressure obtained by analyzing an unflawed ring (no cracks) under the given loads P. In general, crack line stress σ_{yy} may be obtained either by solving the analytical elasticity problem or by performing finite element stress analysis of the unflawed ring. The analytical solution, if available, is of course preferable. For the case under consideration, the stress distribution on the crack line was found by taking the negative of the analytical stress distribution³ obtained for the unflawed ring when P = 1. The node numbers, nodal coordinates and $\pi.\sigma_{yy}$ of the points lying on the crack line are listed below.

Node	X-coordinate(in)	T.σyy(psi)
100 (crack mouth)	-1.0	-10.1365
101	-1.05625	- 7.72739
102	-1.1125	- 5.96154
103	-1.16875	- 4.61541
104	-1.225	- 3.55142

Node	X-coordinate(in)	^π · ^σ yy(psi)
110	-1.28125	-2.68151
115	-1.3375	-1.94734
124	-1.36563	-1.61820
129	-1.39375	-1.30913
138	-1.42188	-1.01697
143	-1.45	-0.739051
152	-1.465	-0.595854
157	-1.48	-0.455750
166	-1.485	-0.409688
171	-1.49	-0.363927
180	-1.4975	-0.295833
185 (crack tip)	-1.5	-0.273276

It is noticed that nodes 100, 101, and 102 belong to the same element whose connectivity is: 81 83 102 100 82 95 101 94 (see connectivity data of Example 2). In accordance with the convention shown in Fig. 1, the nodes 100, 101, and 102 form side 3 of the element. In a similar fashion we can identify the group of nodes (102, 103, 104), (104, 110, 115), (115, 124, 129), etc. being side 3 of respective elements to which they belong.

Next, for each element side, such as (100, 101, 102), we have polynomial traction distribution of the form:

 $T_y = A_1 + A_2X + A_3X^2 + A_4X^3 = \sigma_{yy} n_y = \sigma_{yy}$ $T_x = B_1 + B_2 + B_3Y^2 + B_4Y^3 = 0$

where n_y is the only nonzero component of the outward unit normal. For the present case $B_1 = B_2 = B_3 = B_4 = 0$. Assuming a quadratic distribution of T_y , $A_4 = 0$. Then, for each group of nodes, such as (100, 101, 102), the coefficients A_1 , A_2 , A_3 are found since the coordinates and pressure distribution at each node is known. For the present case this was accomplished by writing a small computer program to solve three linear algebraic simultaneous equations. The result was as follows.

·	~2	A3
51.03588	80.16584	32.35651
28 • 82489	39.99275	14.19187
17.87724	22.03494	6.82783
12.93540	14.57838	4.01532
10.66238	11.31630	2.84494
9.28895	9.41632	2.18785
8.69650	8.61401	1.91623
8.42768	8.25336	1.79527
	51.03588 28.82489 17.87724 12.93540 10.66238 9.28895 8.69650 8.42768	51.03588 80.16584 28.82489 39.99275 17.87724 22.03494 12.93540 14.57838 10.66238 11.31630 9.28895 9.41632 8.69650 8.61401 8.42768 8.25336

Using the displacement boundary conditions shown in Fig. 10b, the SRLOIA data for the problem can now be prepared. Since the mesh and displacement boundary conditions remain identical to those of Example 2, only the changes are described in the following.

a.	Data set	1: NCONC = 0 (no concentrated loads).
ь.	Data set	2: Same as Example 2.
c.	Data set	3: Same as Example 2.
d.	Data set	4: Not present.
e.	Data set	5: Same as Example 2.
f.	Data set	6: NTYPEL = 1 for those elements along the crack surface
g٠	Data set	7: Same as Example 2.
h.	Data set	8: Same as Example 2.
i.	Data set	9:
	3	
	51.03588	80.16584 32.35651 0.0 0.0 0.0 0.0 0.0
	3	
	28.82489	39.99275 14.19187 0.0 0.0 0.0 0.0 0.0
	3	
	17.87724	22.03494 6.82783 0.0 0.0 0.0 0.0 0.0
	3	
	12.93540	14.57838 4.01532 0.0 0.0 0.0 0.0 0.0

3 10.66238 11.3163 2.84494 0.0 0.0 0.0 0.0 0.0 3 9.28895 9.41632 2.18785 0.0 0.0 0.0 0.0 0.0 3 8.6965 8.61401 1.91623 0.0 0.0 0.0 0.0 0.0 3 8.42768 8.25336 1.79527 0.0 0.0 0.0 0.0 0.0 j. Data set 10: Same as Example 2. k. Data set 11: Same as Example 2.

The data image produced by SRLO1A, and the finite element solution were as follows.

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31	51	49	29	39	56	38	30	ú	1	1.00030
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71	91	89	69	79	90	78	70	0	1	1.0000
73	93	91	71	80	92	79	72	ú	1	1.00000
81	83	162	106	82	95	101	94	1	1	1.0000
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163	149	151	165	155	150	156	164	0	1	7.00000
159	173	171	157	167	172	166	158	1	1	1.0000
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NCOL	9	₽ •(\$978	1.77598
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NCOE	11	0.00000	1.08750
NGDE	12	0.09930	1.94375
NODE	13	v	STATUS
NCDE	14	19134	.96194
NCDL	15	21287	1.07016
NCDE	16	23441	1.17838
NODE	17	28702	1,44291
NODE.	18	33963	1.70745
NODE.	19	36116	1.81566
NUUL	20	38269	1.92388
NODE	21	38268	.92368
NODE	22	40421	.97535
NCOL	23	42574	1.(2782
NODE	24	44727	1.07979
NODE	25	40879	1.13175
NODE	26	52141	1.25879
NODE	27	57403	1.38582
NUUL	28	-+ 62065	1.51286
NOUL	29	-+67926	1.63989
NUUE	30		1.69185
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NODE	33		1.84775
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NODE	55	95724	.61621
NODE	56	99898	.66750
NODE	57	-1.22324	.81735
NCDE	58	-1.44750	. 967 19
NCDE	59	-1.53924	1.12049
NCOL	60	-1.63193	1.05979
NODE	61	- 92338	.38205
NODE	62	- 97585	.4:421
NCDE	£3	-1.92782	. 42574
NODE	64	-1. (7979	. 44727
NODE	65	-1.13175	46379
NODE	66	-1.25879	•52141
NCOL	67	-1.73582	. 57403
NCOE	68	-1.51286	. 62665
NCDE	69	-1. 53989	.67926
NODE	70	-1.69186	.7079
NGDE	71	-1.74382	.72231
NCOL	72	-1.79579	.74384
NODE	73	-1.84776	.76537
NGDE	74	95694	.24634
NODE	75	-1.07816	.26787
NODE	76	-1.17338	.28940
NODE	77	-1.44291	.34202
NCOL	78	-1.70745	.39463
NODE	79	-1.81536	•41616
NCOE	80	-1.51338	+ + 37 6 9
NODE	81	39160	.11000
NODE	82	-1.05125	11000
NOOL	63	-1.11250	11008
NODE	84	-1,16875	•11398
NCOE	85	-1.22500	.11000
NCOE	8 E	-1.36251	•11000
NGDE	87	-1.50000	•11998
NODE	88	-1.63750	•118:0
NGDE	89	-1.77500	+11985
NCDE	90	-1.83125	•11606
NCOE	91	-1.03750	•11000
NGDE	92	-1.93375	•11000
NODE	93	-1.98UJ	.11395
NCDE	94	99500	.05500
NODE	95	-1.11250	.05500
NODE	96	-1.22597	.65500
NUUE	97	-1.7750u	.05500
NUUL	98	-1.00750	.05500
NUUL	99	-1.99000	• 65560
NUUL	100	-1.LUUUU	0.0000
NUUE	101	-1.00025	0.00000
NUUL	162	-1.11250	0•6 ~0 36

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NGDL	103	-1.16875	00000.
NODL	104	-1.22500	0.00000
NODE	105	-1.77530	0.00000
NODE	106	-1.83125	6.66666
NUDE	167	-1.88756	0.00000
NODŁ	168	-1.94375	0.00000
NODE	109	-2.00000	0.0000
NODE	110	-1.28125	0.0000
NODE	111	-1.28125	.10000
NCDE	112	-1.50890	.16000
NODE	113	-1.71875	.10000
NODE	114	-1.71875	0.00000
NCDE	115	-1.33750	6.66000
NCDE	116	-1.33750	•ú4500
NODL	117	-1.33750	.09000
NODE	116	-1.41875	09090
NGD	119	-1.50010	.09000
NODE	320	=1.58125	. 0 90 0 6
NODE	121	-1-66250	.09000
NODE	122	-1.00250	64599
NODE	123	-1.66250	0.0000
NODE	124	-1.36563	6.(0036
NODE	125	-1.36563	. [8250
NODE	126	-1.54700	• U 825 Ŭ
NODE	127	-1.63438	.08299
NOJE	128	-1.63438	6.61660
NODE	129	-1.39375	0.00000
NGDE	130	-1.39375	. (3750
NODE	131	-1.39375	.17500
NODE	1 32	-1.44588	.17530
NODE	133	-1.59060	.075(-0
NCOL	134	-1.55313	.(7500
NOOL	135	-1.60625	.17500
NODE	136	-1.69525	•C3756
NODE	137	-1.60025	6.16600
NODE	138	-1.42188	0.00000
NCDC	139	-1.42188	.6250
NGDE	140	-1.50000	.06250
NODL	141	-1.97313	.06250
NCDŁ	142	-1.57813	0.10000
NGĐE	143	-1.45000	6.60.30
NODE	144	-1.45)00	•€25J9
NODE	145	-1.45000	•65009
NODE	146	-1.47509	.15000
NCOL	147	-1.54963	.05000
NCDr.	148	-1.52500	●戶与以前位
NODE	149	-1.55000	.65000
NODL	150	-1.55090	.02500
NCDE	151	-1.55,20	6.(1000
NODE	152	-1.46590	0.0000
NODL	153	-1.46563	.03500
NGÙL	154	-1.50000	•C35v8
NODE	155	-1.53500	.03501

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NODE	156	-1.53500	0.00000
NCDL	157	-1.48300	0.00000
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NODE	159	-1.48000	• 0 2 0 Ŭ O
NODE	160	-1.49000	.02000
NODE	161	-1.50000	•02000
NODE	162	-1.51000	.02000
NODE	163	-1.52000	. (2000
NODE	164	-1.52330	.01986
NODE	165	-1.52000	0.00000
NODE	166	-1.48530	6.0000
NCDE	167	-1.48547	•C1354
NODE	168	-1.50000	•61506
NODE	169	-1.51354	•01354
NODE	170	-1.51500	0.0000
NODL	171	-1.49000	6.66600
NODE	172	-1.49147	·L9354
NODE	173	-1.49293	· U (7 17
NODE	174	-1•49547	•66854
NCDE	175	-1.50000	.01609
NODE	176	-1.50354	.00854
NODE	177	-1.50707	. 6737
NGDE	178	-1.50854	.0354
NCDE	179	-1.51890	0.00000
NGOE	180	-1.49750	0.00000
NODE	181	-1.49823	• 0 • 1 77
NCDE	182	-1.50090	·[L250
NODE	183	-1.50177	•00177
NODE	184	-1.53250	60000.0
NODE	185	-1.50350	0.01000

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SIDE	3COEFF	0F	POLYNOMIAL:	17.87724	22.03496	6.82783	0.10000
				0.00030	9.10000	0.01000	0.10001
SIDE	3COEFF	0F	POLYNUMIAL:	12.93543	14.57838	4.91532	0.10103
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DE	49	-1.25511	1.25511	.7952857E-03	. 2413143 ^E -36
j0£	50	-1.29489	1.29489	14717385-63	• 2298423c-16
DE	51	-1.33466	1.33466	1060J34E-07	.21845982-36
) DE	52	-1.37444	1.37444	2011A71E-07	.2058773E-06
DE	53	-1.41421	1.41421	29562 3FE-67	•1953612 ^E -fio
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0r	55	+- 9.724	.56520	 1830427E-06 	 352ù7162-P6
DE	56	99896	•66750	 1545837E-06 	• 32251255-nf
DE	57	-1.22324	. 81735	• 120894 cE-P6	- 24432195-06
ĴΕ	58	-1.44755	.96719	 79742736-07 	 1693751≦−66
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306	69	-1.63389	• 6 7 9 2 6	. 1733332E-06	20-32266462 .
ä	70	-1.69186	• 7 c t 7 9	. 17299977E-Q6	. 5032858[-07
ΩĘ	71	-1.74382	.72231	 1658594£−₿6 	 39356215-07
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E-06 90-3 33253644-06 · 27487785-06 •1329801E-n6 -- 94100935-09 -.34387355-07 . 46278142-76 .37193535-06 33611615-06 . 3521566E-06 . 26631915-06 .1553771E-06 .61334755-07 1533421E-07 • 7827378E-09 -- 4246841*č*-08 -- 3455361E-98 -- 20593946-97 4157859E-06 - 34598895-06 - 2721314F-06 --7351116E-10 -- 51302596-38 -.97756732-08 • 4293266E-06 • 39368445-06 -3559681c-06 2361128E-36 .3187916-07 -.14418796-07 3849695 5- 64 .3175374 2776387 **.**• • . • • 29515525-36 29726826-86 31J72876-05 2932021E-n6 20551095-06 41432832-06 41049226-06 40521826-65 40313395-06 39554352-06 396-396-36 39886775-05 39355346-06 3967 n 45E-US 39492576-66 39357465-05 3919154E-66 44785176-06 44J9198E-05 43757696-06 40934846-09 46197495-06 397286FE-06 4821429E-06 47850155-06 4759628E-05 47604275-05 4757115E-06 40-3024604 4067433E-06 46403105-06 40108446-06 39859445-05 32472915-96 4174814E-06 4754245E-06 34311145-0 .11040 26787 69224* .11100 .1100C · : : : • • • .34232 .39463 .41616 • 11001 • 7 7 6 6 6 .11306 .111 uľ .11000 .11076 .11900 .05533 • 055 3 • • 65536 .05500 100000.0 9.6000 10000-0 000000 C. C P G G C 0.66706 5. L 86 30 .28940 .1116 .110ⁿč •110JE . 15511 .55505 2.66306 00000-1 0.00000 24634 -1.1125n -1.7.15 -1.17833 -1.75745 -1.615hř -1.91338 -1.05125 -1.11250 -1.16875 -1.5000 -1.c3750 -1.77500 -1.63125 -1.48750 -1- 43375 -1.53000 0[265 ---1.1250 -1.77560 -1. 28750 100065-1--1. P(. 2. -1.05625 -1.16375 -1.22506 -1.77530 -1.83125 -1.94375 -1.44291 -1.2506 -1.225-5 -1-88750 -1.28125 93966 ---2. 1 1 1 -- 95691 -1.3025 э 6 5 55 102 103 104 105 108 169 83 95 150 101 106 117 116 52 20 2 28 29 ů 19 20 84 35 36 87 88 30 91 95 46 5 97 35 NODE NCOE ACON NODE NODE NODI NCDE NOUE NODE NOUE NCUE NCOE NCDE NCDE NOUL NODE NCDE NCUL NODL NCDC NCOL NCDE NCOL NODY NCDE NCOE NCDL NCDL NCDL NODL NCON NODN JCON NCOL NODE ACO V

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• 5132596E-07	• 4C37395E-US	.17505	-1.50968	
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The above results show that K_I for the problem is 2.60167 psi \sqrt{in} . The same problem of double notch ring compression specimen when solved by applying concentrated load P directly (Fig. 10a) gave stress intensity factor value of 2.59355 psi \sqrt{in} . The 0.3% difference in the two values could be attributed to round-off error in the calculation of nodal stresses from elasticity solution and in the evaluation of pressure polynomial constants. Further, the assumption of quadratic pressure distribution over each element side may have contributed to the difference. An interesting exercise would be to try constant, linear, and cubic pressure polynomials.

4. EXAMPLE 4: COMPACT TYPE SPECIMENS

One of the advantages of using programmed mesh generation along with the finite element program is that changes in geometry pose no difficulty. For a compact tension test specimen (Fig. 11) of standard geometry (H/W = 0.6) and for WOL specimen (H/W = 0.486) SRLO1A results can be found in reference 4. For the present example we choose the following dimensions:

H = 1.2 in.
W = 2.4 in.
W₁ = 3.05 in.
S = 0.0938 in.
F = 1.417 in.
E' = 0.55 in.
R = 0.25 in.
θ = 40.0 deg.
A = 1.2 in.

The mesh generation program SRLCMP (Appendix I) was executed using the following cards:

Job Card ATTACH,LGO,SRLCMP,ID=M760328. ATTACH,TAPE8,SRLCM,CY=2,ID=M760328. LGO. End-of-record card 1.2 0.55 1.417 1.2 0.0938 40.0 2.4 3.05 0.25 0.55 End-of-job Card

The last entry on the data cards is the distance n - n (Fig. 11). This distance should always be kept less than $2 \times E'$ and larger than S. The location of point n can be chosen to correspond with the actual COD measurement location in an experimental set-up.

The use of plotting program SRL11 provided the mesh plots shown in Fig. 12. Due to symmetry about the crack axis, only the upper half of the geometry is used in the analysis. The displacement and force boundary conditions are shown in Fig. 13. The crack tip is fixed in both X and Y directions to eliminate rigid body motion.

The data image produced by SRL01A was as follows:

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106	1	1		0.0	00000	0000	000		Ö.	000	00	0000000
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108	1	1		0.0	00000	0000	000		0.	000	00	0000000
104	· · · O - ·	- 1	 .	0.0	0000	0000	000		01	000	100	0000000
100	0	1		0.0	00000	0000	000		0.1	000	00	0000000
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128	-156	154	118	142	155	141	119			
122	158	156	120	143	157	142	121	Ĩ	1	1.00000
124	-16-0	-158	122	144	159	143	123	1		1.00000
126	162	160	124	145	161	144	125	1	1	L 1.00000
150	173	-171	-148	-164	17-2	163	. 1 49			
152	175	173	150	165	174	164	151	1	1	L 1.00000
154	· 177	175	152	166	176	165-	153			L1-00000
156	179	177	154	167	178	166	155	1	1	1.00000
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160	183	181	158	169	182	168	159	1	1	1.00000
162	185	183	150	170	184	169	151	1	1	1.00000
173	196	194	171	187	195	186	172	1	1	1.00000
175	198	-196	-17.3	199	-197	187	174			
177	200	198	175	189	199	185	176	1	1	1.00000
179	202	200	177	190	201	-1-89	178	••••••		
181	ZU4	202	179	191	203	190	130	1	1	1.00000
183	· 206	-204	-141	-192	205	· 191 ·	-132-			1 1,00000
185	200	205	183	193	207	192	154	1		L 1.00000
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198	221	219	136	211	220	210	1 37	1	1	L 1.00000
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263	2.7	225	243	241	200	293	262	4				0 0 C
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وأنفته ويعالفهم فالمنازع فالمستعاقة المرديم والمعادمة والمستقدم والألي وليردده

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.1989700E+00	NODE	7
-2369858E+08-	NODE	
.2750000E+00	NODE	9
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.5500000F+00	NODE	11
.7750000E+00	NODE	12
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.16095002+00	NODE	22
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•15000005401	4JU:	30
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+ 4070000E-01 1220700E+00	1005	40
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والأقلاف والأقافة فالمستعاد مستعادهم ومرارية المتكلا وتركامه والمتريبين

والمقافع والأقلاق ومعافلا فالمعامل والمرون كروحا فالملاحم والمستخدم والمسافرة

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•5500000E+00	NODE	57
-7750000E+00	NODE	-58
•1000000E+01	13 DE	59
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.120000E+01	NODE	61
Ð.	-NO DE-	-62
-4690000E-01	NO DE	63
-1229300E+00	-1385-	64
-1989700E+00	NODE	65
-2750000F+00-		-66
-5500000E+00	NODE	67
-1200000F+01	NODE	69
-A	MANE	
2345000F-01	NODE	71
	-4902	-79
. 8491500E-01	NUDE	72
-1609500E+00	NODE	75
23698505100	1000	77
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0		-N38E-112-
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0.	.8491500E-01	NODE 115
		-130E-116
0	.1609500E+00	NODE 117
A		-NODE -148-
0	-23693505+00	NODE 119
	2750000E400	-120-
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Q	<u>5500005400</u>	NO.05 122
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Δ	1000005401	NO. 124
	- 11000000000	NODE 125
	12000000000	NODE 129
- 77500005-02		NODE 127
	.26500006-02	NODE 127
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		NUUL 131
		4006 132
19500UUE-U1	U.	NJUE 133
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•.2945500E=01		
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30000D0E-01	•4690000E-01	ND DE 139
39900 00E-01 · ····· · ·····		-43-01-140
300000E-01	• 1989/ UUE+UU	NODE 141
		-+30E-142
300000E-01	.5500000E+00	NDDE 143
300000E-01	.100000E+01	ND DE 144
3000000E-01	•1200000E+01	NODE 145
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3848500E-01	.3193500E-01	NODE 147
600000E-01	0.	NO DE 148
6000000E-01	•2345000E-01	NDDE 149
600000E-01		130E 150
600000E-01	•8491500E-01	NODE 151
600000E=01	•1229300E+00	NODE 152
600000E-01	•1609500E+00	NODE 153
•.600000E=01	-1989700E+00	NO 01-154
6000000E-01	•2369550E+00	NDDE 155
600000E-01		NO DE 196
600000E-01	•4125000E+00	NO DE 157
GOOROOE-01		NODE 158
6000000E-01	•7750800E+00	NODE 159
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.1100000E+01	NO DE 161
	-162 -
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.4690000E-01	-N30E-164
•1229300E+00	NO DE 165
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•2750000E+00	NDDE 167
	NO DE 160
	NJUL 109
•1200000E+01	NODE 170
.4690000E-01	NODE 173
	NO OF 176-
1229300E+00	NODE 175
-1609500E+00	-1305 176
.1989700E+00	NODE 177
	NODE 178
.2750000E+00	NODE 179
	- 438E-188
.5500000E+00	NODE 161
	NODE-182
•1000000E+01	NJDE 183
	NODE 485
• 12000002+01	NJUE 105
46000005-01	NODE 187
.1989700F+00	NODE 189
.5500000E+00	NODE 1º1
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1200000E+01	NDDE 193
- 0	1905-194
.2345000E-01	NO DE 195
•0491500E-01	NODE 197
• 1509500E+00	NJUL 193 NDD= 200
23698505400	NODE 200
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.4125000E+00	NDDE 203
.7750000E+00	NODE 205
1100000E+01	NODE 207
-1200000E+01-	490E- 203
0.	EDS EDCK
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.1229300E+00	NJDE 211
	-1000 212
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. 1000000F+01	NONE 215

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833150	0E-01
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-1609500E+00-	1302	222
.1989700E+00	NODE	225
	- 400E -	224
• 273UUUUCTUU	NODE	226
550000C+00-	NODE	227
	N305	224
-100000E+01	NODE	229
-1100000E+01	- 43DE	230
.1200000E+01	NODE	231
.4690000E-01-	-NODE	232
.1229300E+00	NODE	233
-19897-00E+00 -	- NO 05	234
.2750000E+00	NDDE	235
-5500000E+00-	-N98E	236
.1000000E+01	NODE	237
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- 1229300F+00	NADE	240
<u>-16095002+00</u>	NODE	262
-1989700E+00	NODE	243
.2369850E+00-		244
.2750000E+00	NDDE 2	245
+4125000E+00-	-43 86-1	246
•5500000E+00	NODE	247
-7750000E+00-	ADE .	248
•1000000E+01	NODE	249
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•1200000E+01	430E 0	251
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-1069700E+00	NODE A	255 255
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-3241250E+00-	-43.05	256
.5500000E+00	NODE	257
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-1100000E+01	10 DE -	306
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The results of the analysis are summarized below:

 $K_{I} = 6.59341 \text{ psi}\sqrt{\text{in}}$ $\delta_{m-m} = 66.43846 \times 10^{-7} \text{ in}$ $\delta_{1-1} = 43.76998 \times 10^{-7} \text{ in}$ $\delta_{p-p} = 48.98702 \times 10^{-7} \text{ in}$ $\delta_{g-g} = 46.30266 \times 10^{-7} \text{ in}$

The points m, 1, p, and g are indicated in Fig. 11 and these correspond to nodes 318, 261, 277, and 281 on the mesh respectively. The displacement values shown above were obtained by multiplying the V-displacements of the corresponding points by two.

5. EXAMPLE 5: THREE POINT BEND SPECIMENS

The geometry of a three point bend specimen is shown in Fig. 14. The data for this problem was generated by using the program of subsection 4 of Appendix I, and the following dimensions:

L = 4.25 in., P' = 0.0625 in. S = 4.0 in., THETA = 40.0 deg. W = 1.0 in., G = 0.2 in. C' = 0.375 in., A = 0.5 in. N = 0.25 in., H = 0.125 in. B = 1.0 in.

The data image produced by SRL01A was as follows:

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	15	13	62	27	14	26	43	1	1	1.	
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63	36	32	61	51	33	59	62	1	1	1.	
65	36	34	63	52	35	51	64	1	1	1.	
67	38	36	65	53	37	52	66	1	1	1.	60000
69		38	67	54	39	53	68	1	1	1.	00000
71	42	40	69	55	41	54	70	1	1	1.	00000
73	44	42	71	56	43	55	72	· 1	1	1.	
71	6 66	44	73	57	45	56	74	1	1	1.	00000
77	48	46	75	58	47	57	76	1	1	1.	
96	61	59	88	79	60	78	89	1	1	1.	00000
92	63	61	90	6.0	62	79	91	1	1	14	00000
94	65	63	92	81	64	81	93	1	1	1.	
96	67	65	94	62	66	81	95	1	1	14	
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135	106	104	133	116	105	115	134	1	1	1. 00090
148	119	117	146	137	118	136	147	1	1	1200000
150	121	119	148	138	120	137	149	1	1	1488898
152	123	121	159	139	122	138	151	1	1	1. 10800
154	125	123	152	14.0	124	139	153	1	1	1208800
156	127	125	154	181	126	140	155	1	1	1.00800
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168	131	129	154	163	138	142	159	-	4	1.00000
162	133	131	168	164	132	143	161		1	1469866
164	136	133	162	466	131	144	163	i	-	1.00000
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103	124	176	101	103	170	100	102	-	4	1 00000
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103	105	190	10/	1/2	199	1/1	100	1	1	7544444
191	102	160	189	173	161	172	190	1	1	160000
193	104	162	191	174	163	1/3	192	1	1	1.00000
203	179	1/7	243	195	178	194	204	1	1	100000
287	181	179	245	196	108	195	206	1	1	1432790
209	183	181	287	197	182	196	208	1	1	1200000
211	185	183	209	198	184	197	210	1	1	140000
213	187	185	211	199	186	198	212	1	1	1.00000
215	189	187	213	280	188	199	214	1	1	1000000
217	191	189	215	201	190	200	216	1	1	1.08000
219	193	191	217	202	192	201	218	1	1	1260880
231	205	283	229	221	204	220	230	1	1	1.00000
233	207	205	231	222	206	221	232	1	1	1 <u>#</u> 00000
235	289	287	233	223	288	222	234	1	1	1.00000
237	211	209	235	224	210	223	236	1	1	<u>140000</u>
239	213	211	237	225	212	224	238	1	1	1. 80000
241	215	213	239	226	214	225	240	1	1	1200000
243	217	215	241	227	216	226	242	1	1	1,00000
245	219	217	243	228	218	227	244	. 1	1	1.00000
257	231	229	255	247	230	246	256	1	1	1.00000
259	233	231	257	248	232	247	258	1	1	1,00000
261	235	233	259	249	234	248	260	1	1	1 .00000
263	237	235	261	258	236	249	262	1	1	1.00000
265	239	237	263	251	238	250	264	1	1	1400000
267	241	239	265	252	248	251	266	1	1	1.00000
269	243	241	267	253	242	252	268	1	1	1.00000
271	245	243	269	254	244	253	27 0	1	1	1,888888
281	259	257	279	273	258	272	280	1	1	1.80800
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•1786335E+81	NODE	50
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•1708335E+01	NODE	53
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-14166785+81	NODE	59
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-1416670E+01	NODE	63
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A14100/82+81	NUCE	75
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-49375000E-01	•1086008E+\$1	NODE	95
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•2916650E+00	•100000E+01	NOCE	103
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- 49375888E-01	5833304E+00	NOTE	124
6 25 COODE-01	.5833300E+00	NOCE	125
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3751104E+14	NODE 149
3750000E+88	NOCE 150
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3750000E+00	NOCE 152
-3750000E+08	NOCE 153
3750800E+10	NOCE 154
3756000E+08	NOLE 155
3750000E+00	NOCE 156
3750000E+00	NOCE 157
3758884E+68	NODE 158
375000 BE+09	NOCE 159
3750000E+00	NOCE 160
.3750008E+90	NOCE 161
3758808E+88	NODE 162
. 375888 QE+88	NODE 163
3750000E+80	NOCE 164
,2812500E+00	NOCE 165
281250QE+06	NOCE 166
.2812508E+88	NODE 167
2812500E+10	NOCE 168
2812500E+80	NOCE 169
.2812500E+00	NOCE 178
,2812 588E+88	NOCE 171
.2812508E+00	NODE 172
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1875090E+00	NOCE 175
.1875868E+88	NOCE 176
1375000E+00	NOCE 177
• 187500 DE+ (0	NOLE 178
1875000E+00	NOCE 179
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#1437588E+40	NOCE 200
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●1437588E+88	NODE 212
£1437500E+00	NOCE 213
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#1437588E+88	NODE 216
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-1437508E+00	NODE 244
-1437500E+00	NOCE 210
-17187885188	NOUE 219
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12187585464	NULE 221
+10101002+88	NOCE 222
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a1218759E+60	NOCE 224
+1218759E+88	NOCE 225
41218758E+80	NOTE 226
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.1218750E+00	NOTE 228
11009664E+88	NODE 229
.100000E+48	NOTE 276
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-10808045+88	NODE 232
-18088885188	NOUE 235
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+023000E-01	NUCE	277
• 6250000E-81	NOUE	200
• 6259809E-91	NOCE	201
6250000E-11	NUCE	262
.6250000E-01	NOCE	263
•6250000E-81	NOCE	Z64
6250888E-81	NODE	265
₿6250000E-81	NOCE	266
•6250000E-01	NOCE	267
•6250000E-01	NOCE	268
.625000\$E-\$1	NODE	269
•6250000E-01	NOCE	271
£6250009E-01	NOGE	Ż71
•3125000E-01	NOCE	272
.3125000E-01	NODE	273
.312540DE-01	NODE	274
-3125008E-91	NOCE	275
-312500 BE-01	NOCE	276
-3125000E-01	NOTE	277
-3125808E-11	NOTE	278
	NOCE	279
8.	NOLE	280
0.	NOCE	281
R.	NODE	282
â.	NOCE	243
	NACE	284
	NOCE	285
u e A.	NOCE	203
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• 4087984E - U1	NOUE	292
● 4687500E=01	NULE	293
#4687500E-01	NOCE	294
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V •	NODE	296
•1562500E-01	NOCE	297
• 3125070E-01	NOCE	298
• 3125000E- J1	NOCE	299
• 312 560 DE - 01	NOCE	300
•3125409E-81	NODE	381
• 3125030L- 81	NOCE	J9 2
•1562500E-01	NOCE	383
0.	NORE	784

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6250000E- 82	NOCE 314
.1250600E-81	NOCE 312
.1250000E-01	NODE 313
•1250808E- {1	NODE 314
-1250000E-01	NODE 315
-1250000E-01	NODE 316
•6250 <b>0</b> 00E-02	NOCE 317
Dei	NODE 318
8.	NOGE 319
•8460019E-92	NOCE 320
• 937 500 0E- 02	NOCE 321
• 5460000E-02	NOCE 322
8.	NOCE 323
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+ 221 989 9E-92	NOCE 325
+4424400E-02	NOCE 326
	NODE 327
	NODE 328
47333999 <u>2</u> -92	NUCE 329
- 221 BBBBE- 82	NUCE 330
B* ##5504045-85	NULE 331
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-1180000F-02	NULE 333
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In this case loads of 0.5 lbs. are applied at points 1 and p (Fig. 14), and the crack tip is fixed in both directions. Therefore the total load point displacement is the sum of the magnitudes of vertical displacements of points p and 1. In Fig. 14, G represents the location of clip gage to measure the crack opening displacement. The result of the analysis gave the following values of stress intensity factor, load point displacement and the crack opening displacement:

 $K_I = 10.6892 \text{ psi}\sqrt{\text{in}}$ Load point displacement = 58.01097 × 10⁻⁷ in. Crack opening displacement = 16.02759 × 10⁻⁷ in.

On the mesh points p and 1 correspond to nodes 30 and 290, respectively. Crack opening displacement is twice the Y displacement of node 229.

6. EXAMPLE 6: SINGLE NOTCH RING SPECIMEN

Single notch specimens in the shape of circular rings are shown in Fig. 15. The program of subsection 2 of Appendix I generates the data for single notch ring compression specimen (Fig. 15a). The data for the tension case is generated by changing the loading condition.

Choosing A = 0.5 in.,  $R_1 = 1.0$  in and  $R_0 = 2.0$  in., the mesh generation program was used to produce the required data for SRLO1A. The data image produced by SRLO1A, which corresponds to the mesh of Fig. 16 was as follows.

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1913081E+(1	.4651850E+98	NODE160
99 00 0 00E + 90	.1650000E+00	NODE161
10725 00E+01	.1650000E+00	NOPE 162
1155 0 00E +01	.1650000E+00	NODE163
1237500E+01	-1650900E+06	NODE164
13200 00E+01	.1650000E+00	NODE 165
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14350 00F +01	-1650000E+00	NOPE 167
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- 12500 ROF +01	0.	NODE 183
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- 1666670F+01	0.	NOBE 185
-, 17500 00F+01		NOCETAR
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- 1371670E+01	.64165006-01	1005 105
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1429170E+01	0.	NONE 209
1429170E+01	• 3541500E-01	<b>NODE 210</b>
1429170E+01	•7083000E-01	NODE 211
1464585E+01	•7083000E-01	NODE 212
1549848E+81	.7063000L-01	10 JE 213
1535415E+01	•7083000E-01	NODE 214
15708 30E+01	•7083000E-01	NODE <u>215</u>
15708 30E +01	.3541500E-01	<b>NODE 216</b>
1570 8 30 E+01	Ũ	100E 217
14387 50E+01	0.	10 PE 218
1433750E+01	.6125000E-01	NODE 219
1500000E+01	.6125000E-01	NODE 220
1561250E+01	•6125000E+01	NO DE <u>221</u>
1561257E+01	0.	100E 222
1448330E+01	0.	NDDE 223
1448330E+01	-2583500E-01	103E 224
1448330E+01	•2167000E-01	NODE 225
1474165E+61	•5167900E-81	NODE 226
1500000000+01	•2167000E-01	10 DE 227
15256 35E+01	.5167000E-01	100E 228
1551670E+01	.5167000E-01	10JE 229
1551670E+01	.258351 E-1	NOPE 230
15516702+01	0.	100E 231
14608 30E+01	0.	<b>NODE 232</b>
14608-30E+01	.3917000E-01	<b>NDDE 233</b>
15 00 00E+01	.3917000E-01	NODE 234
153917}E+01	.3917068E-01	1005 235
1539170E+01	0.	13nE 236
1473330E+01	0.	NODE 237
1473330E+01	.1333500E-01	101E233
1473330E+01	•2667%LLE-01	10PE 239
1486665E+01	.2667000E-01	100E240
1500000E+01	•2667000E-01	NODE 241
1513335E+01	.2667000E-01	<b>NODE 242</b>
1526670E+01	.2667000E-01	100E243
1526673E+11	•1333500E-01	NODE 244
1526670E+01	0.	13DE 245
1481665E+01	0.	103E246
1483130E+01	•1687000E-01	100E247
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1516870E+01	.1687008E-01	1002249
15 18 3 35E +01	0.	NODE 250
1490 0 COE +01	0.	100E251
1491465E+01	.3535000E-'2	VODE252
14929308+01	•707000E-02	10 NE 253
1496465E+01	.8535000E-02	1002254
-, 15000 00E+01	.100000E-01	VOPE255

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1503535E+01	.8535000E-02	NO DE 256
1507070E+01	.707JJILE-02	103E257
1508535E+01	.35350 00E-02	NDPE258
1510000E+01	D	NODE259
14375 00E+01	U •	NODE260
1496230E+01	•177505bE-#2	NODE261
1500000E+01	• 2500000E-02	NODE 262
15 <b>0</b> 1770E+01	.1770000E-02	NODE 263
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15552 00E+11		NJDE266
1590000E+01	0	NOCE 267
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The execution of SRLO1A resulted in the following values for stress intensity factor, crack opening displacement, and load point displacement:

$$\begin{split} & K_{\rm I} = 1.8940 \ {\rm psi}\sqrt{\rm in} \\ & {\rm COD} = 2 \times ({\rm Y-displacement} \ of \ node \ 180) = 6.45932 \times 10^{-7} \ {\rm in.} \\ & \delta_{\rm L.P} = {\rm sum} \ of \ {\rm the} \ absolute \ {\rm value} \ of \ {\rm X} \ - \ {\rm displacements} \ of \ node \ 189 \\ & {\rm and} \ node \ 13 = 15.31517 \times 10^{-7} \ {\rm in.} \end{split}$$

# SECTION VI

## CONCLUSION

The report contains user's manuals for the following programs:

- 1. SRLO1A: A two dimensional linear elastic finite element analysis code for crack problems.
- 2. SRL11: A mesh plotting program.
- 3. SRLC: A mesh generation program for C-shaped and doublenotch ring shaped specimens.
- SRLRNG: A mesh generation program for single-notch ring shaped specimens.
- 5. SRLCMP: A mesh generation program for compact tension type specimen geometries.
- 6. SRLBND: A mesh generation program for bend specimen geometry.

Although the mesh generation programs have been carefully tested, the user should not expect these programs to produce acceptable data for arbitrarily chosen geometrical dimensions for any given specimen shape. The programs are however expected to generate proper data for standard geometries with minor modifications.

It is highly recommended that a new user try to solve one or more of the illustrative examples presented in Section IV before attempting to solve a new problem.

### APPENDIX I

#### MESH GENERATING PROGRAMS

#### 1. C-SHAPED SPECIMEN

The finite element data for SRLO1A to analyze a C-shaped specimen may be generated by executing the following program:

Job Card ATTACH,LGO,SRLC,ID=M760328. ATTACH,TAPE8,SRLC1,ID=M760328. LGO. End-of-record N (Total number of data sets to be generated minus 1) A RI RO (Crack length, inner radius, outer radius) . . . . . A RI RO End-of-job.

- Note: a) Format for N is (I4)
  - b) Format for A, RI, RO is (3F10.5)
  - c) The mesh may also be used to analyze double notch circular ring specimens (see examples 2 and 3).

2. SINGLE-NOTCH RING SPECIMEN

وأفرده فتحرف فالمكام فخفاهما كالألاحت معروقها فستقرد وشرعانهم

Data for a single notch ring specimen (Fig. 15a) can be generated by using the following program.

Job Card ATTACH,LGO,SRLRNG,ID=M760328. ATTACH,TAPE8,SRLRG,CY=2,ID=M760328. LGO. End-of-record A RI RO (crack length, inner radius, outer radius) End-of-job.

Note: a) Format for the data card is (3F10.5)

- b) For multiple runs put as many LGO. cards as number of cases, and place an end-of-record card in front of each data card.
- c) The mesh may also be used for a single notch ring tension specimen (Fig. 15b) simply by changing the location of the applied load from nodes 13 and 189 to node 81.

3. COMPACT TYPE SPECIMEN

The data to analyze a compact specimen of given geometry may be generated by using the following program.

Job Card ATTACH,LGO,SRLCMP,ID=M760328. ATTACH,TAPE8,SRLCM,CY=2,ID=M760328. LGO. End-of-record A E F H S THETA W W1 R GS End-of-job

Note: a) Format for the data card (10F7.4)

- b) GS is the distance n-n in Fig. 11.
- c) Procedure for multiple runs is the same as for single notch ring specimen of the previous section.

4. THREE POINT BEND SPECIMEN

```
Job Card
ATTACH,LGO,SRLBND,ID=M760328.
ATTACH,TAPE8,SRLB,CY=2,ID=M760328.
LGO.
End-of-record
L S W G'N P'THETA G A H B
End-of-job
```

- Note: a) Format for the data card (11F7.4) for definition of symbols see Fig. 14.
  - b) For multiple runs follow the procedure described in Section3 for single notch ring specimen.

## APPENDIX II

### PLOTTING PROGRAM

## 1. USER'S GUIDE FOR MESH PLOTTING PROGRAM (SRL11)

a. Data Set 1, Format (314), Number of cards = 1

Columns	Variable/Constant	Definition
4 5-8	6 NPOIN	Output unit Total number of nodes
9–12	NELEM	* Total number of elements

*See a. Section III.

b. Data Set 2^{*}, Form (I4,2F10.5), Number of cards = NCPOIN^{**}

Columns	Variable	Definition
1-4	I	Node number of corner point
5-14	X(I)	Y-coordinate of I
15-24	Y(I)	Y-coordinate of I

* Use the same data cards as 5. Section II. ** See a. Section III.

c. Data Set 3, Format (not applicable), Number of cards = 1

(Blank Data Card)

d. Data Set 4, Format (814), Number of cards = NELEM *

Columns	Variable	Definition
1-4	I	Element connectivities. Use Data Set 6 of f., Section III.
5-8	I + 1	Element connectivities. Use Data Set 6 of f., Section III.
9–12	I + 2	Element connectivities. Use Data Set 6 of f., Section III.
13-16	I + 3	Element connectivities. Use Data Set 6 of f., Section III.
17-20	I + 4	Element connectivities. Use Data Set 6 of f., Section III.
21-24	I + 5	Element connectivities. Use Data Set 6 of f., Section III.
25-28	I + 6	Element connectivities. Use Data Set 6 of f., Section III.
29-32	I + 7	Element connectivities. Use Data Set 6 of f., Section III.

* See a. Section III.

# e. Data Set 5, Format (315, A10), Number of cards = 1

Columns	Variable/Constant	Definition
5	1	
6-10	12	
11-15	L	Output parameter
16-25	LABEL	MESH TITLE
		1





#### Instructions

L	Give 1 if elements and nodes are not to be numbered.
	Give 2 if only elements are to be numbered.
	Give 3 if nodes and elements are to be numbered.
	Give 4 if only nodes are to be numbered.
LABEL	Give ten character title (including spaces, such as RING SPMN1).

NOTE: Only the corner nodes of the mesh are numbered by the program.

f. Data Set 6, Format (8F10.5), Number of cards = 1

Variable/Constant	Definition
YMIN	Minimum of Y-coordinates of all nodes
YMAX	Maximum of Y-coordinates of all nodes
XMIN	Minimum of X-coordinate of all nodes
XMAX	Maximum of X-coordinates of all nodes
YSCALE	Scale on the Y axis
XSCALE	Scale on the X axis [*]
1.0	
1.0	
	Variable/Constant YMIN YMAX XMIN XMAX YSCALE XSCALE 1.0 1.0

* It is recommended to use SCALE = YSCALE = Plot dimension desired divided by actual mesh dimension. Plot dimension desired is limited by the size of the paper available at CALCOMP plotter. Usually, Y dimension of the plot = (YMAX - YMIN) × YSCALE should not exceed 8 inches.

g. Data Set 7, Format (not applicable), Number of cards = 1

(Blank Data Card)

2. SAMPLE JOB SET-UP FOR THE BEAM PROBLEM OF SECTION II, 5.

AAA, CM150000, T100, I0100, STANY. M760328 ATTACH, CCAUX, CCAUX, ID=LIBRARY, SN=ASD LIBRARY, CCAUX. ATTACH, LGO, SRL11, ID=M760328. LGO.

End-of-record card.

6	18	3
1	4.0	0.5
3	4.0	1.5
6	3.5	0.5
8	3.5	1.5
11	1.5	0.5
13	1.5	1.5
16	0.5	0.5
18	0.5	1.5

### BLANK CARD

1	3	8	6	2	5	7	4
8	13	11	6	10	12	9	7
16	11	13	18	14	12	15	17
	1	12	3	BEAM			
0.5	1.5	1.0	.0	2.0	2.0	1.0	1.0
В	LANK CA	RD					
Rad-	of_dete	oard					

#### REFERENCES

- 1. K. J. Bathe and E. L. Wilson, <u>Numerical Methods in Finite Element</u> <u>Analysis</u>, Prentice-Hall, 1976.
- 2. S. R. Barsoum, On the Use of Isoparametric Finite Elements in Linear Fracture Mechanics, Int. J. Num. Meth. in Engrg., <u>10</u> (1), 1976.
- 3. N. E. Ashbaugh and J. Ahmad, paper submitted to the Int. J. of Solids and Structures.
- 4. N. E. Ashbaugh, Mechanical Property Testing and Materials Evaluation and Modeling, AFML-TR-79-4127, Air Force Materials Laboratories, Wright Patterson Air Force Base, OH, 1979.

SIDE 2: (I+I)-(I+2) SIDE 3: (I+2)-(I+3) SIDE 4: (I+3) - (I)Y I I+4 I+7 I+1 I+3 6+1 I+5 I+2 Х

SIDE 1: (I)-(I+1)

Figure 1. Eight Noded Quatrilateral Element.



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Figure 2. Special Forms of General Eight Noded Quadilateral.



Figure 3. Cantilever Beam.





(Ь)





(d)



Second Street and an and a second street and the







Figure 5. Mesh Plotted by SRL11. (Beam)


(a)

وأحمدها فالمتكاف المحمد والمتحد والمتحدية

(Ь)

Figure 6. (a) Semi-Circular Cracked Ring. (b) Region Needed for Analysis.



Figure 7. Mesh for C-Specimen.





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N Car States





 $R_i = 1.0 IN.$  $R_0 = 2.0 IN.$ A = 0.5 IN.

Figure 9. (a) Double Notch Ring in Tension.

(b) One Quarter of the Ring with Displacement Boundary Conditions.



Sector Sector

1000

ومناور ومحافظة فالمتحدث فالأواملة فلأنفظ ومعترض والمحاري ومراجع المحاولة فتحوز والمتحد والمراجع المحافظ والمعام

Sec. Sec.

فالتفاقيني والماهمات فأنقادها والمتعارية إليان مدغاتها ومحتانا فالمناقب معانيتها ويورد مماكم فتغارب لإلار يتعادين مر







Figure 10. (a) Double Notch Ring in Compression.

(b) One Quarter of the Ring with Crack Line Pressure and Displacement Boundary Conditions.



Figure 11. Compact Specimen. (Schematic)



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Figure 12 (a). Mesh for Compact Specimen. Upper Half of Specimen.



Figure 12 (b). Mesh for Compact Specimen. Details of Crack-Tip Region.

17 C 12 C



POISSON'S RATIO = 0.3 THICKNESS = 1.0 IN.

وتستأد المتناد المروي والمرود والمروس





Figure 14. A Three Point Bend Specimen, (Schematic)



(a)

فليقدم والمتحافظ معتمد والمناسبة المحافظ

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Figure 15. Single Notch Ring Specimens.

- (a) Tension
- (b) Compression

