# Fixed-End Moment Equations for Continuous Prestressed Concrete Beams 

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## SYNOPSIS

Fixed-end moments created by prestressing for constant cross-section beams are given. These fixed-end moments may be used with structural analysis methods to design continuous prestressed concrete beams. Fixedend moments for cable profiles composed of straight line segments or parabolic segments are presented for both exterior and interior spans. Only symmetrical cable profiles are given for interior spans. Tables are presented for aiding in the solution of difficult equations.

## CONTINUOUS BEAMS

In recent years there have been many excellent papers written about prestressed concrete including the design and analysis of both simple and continuous prestressed concrete beams. Some of these papers were written by Parme and Paris ${ }^{1,2}$, Fiesenheiser ${ }^{3}$, Moorman ${ }^{4}$, and Lin $^{6}$.
$\mathrm{Lin}^{6}$ has presented a very interesting and useful method in which he balances the dead load effect on the beam with the imposed load of the prestressing forces. This method eliminates the main difficulty in designing continuous prestressed concrete beams. Hence, the load balancing method is recommended in instances where it can be used. In cases where the dead load either is not or cannot be balanced by the prestressing force, the

[^0]following equations and tables are submitted for use.
A brief description of continuously prestressed concrete beams will be given. The assumption is made that the reader is familiar with the design and analysis procedures of a simple prestressed concrete beam, namely, a prestressed concrete beam which is simply supported at each end, not continuous over a support. It is also assumed that the reader is familiar with the ordinary analysis of statically indeterminate beams.

Assumptions which are usually made in both simple prestressed concrete beams and continuous prestressed concrete beams are given in Moorman's paper ${ }^{4}$. These assumptions are

1. Hooke's Law is valid
2. The principle of superposition holds
3. The horizontal component of the tension in the cable is equal to the tension in the cable
4. The friction force is negligible
5. The lateral force from the
cable is either vertical or normal to the axis of the member
6. the loss of cable tension due to creep is negligible
7. The reduction in cross-sectional area because of the cable ducts is negligible
In simple prestressed concrete beams, the moment induced by the prestressing force is equal to the horizontal force in the cable times the distance of the cable from the centroid of the beam. Elastic rotation at the supports and elastic deflection along the beam can occur without creating any moments at the supports because of the hinged ends of the beam. The term hinged-end refers to a beam whose end is free to rotate without moments being induced by this rotation. When the dead load and live load of the beam is not considered, the line of pressure from the prestressing force, or stress on the concrete, coincides with the cable profile.

In a continuous prestressed concrete beam without dead load or live load, the line of pressure on the concrete may or may not coincide with the cable profile. If the line of pressure on the concrete does coincide with the cable, the cable is known as a concordant cable. In many cases the line of pressure on the concrete will not coincide with the cable profile. The reason for this is as follows:

Moments from the prestressing cable are induced along the continuous prestressed concrete beam as they are in the simple prestressed concrete beam. These moments along the beam cause deflections and in turn rotations at the supports. These rotations are not free to take place but are restrained due to the continuity of the beam. The resistance to these rotations cre-
ates moment at the interior supports of the beam. These moments at the support induced by the continuity of the structure cause the center of pressure on the concrete to deviate from the location of the cable profile.

Fig. 1(a) shows an unloaded end span of a continuous beam with $M$ the moment created by the beam restraint at the interior support. The moment, $M$, creates an extra reaction at each end of the exterior span. This reaction is really due to the force in the cable and the continuity of the structure. The extra reaction gives a moment curve which varies linearly between supports as shown in Fig. 1(b). As a result the center of pressure on the concrete varies linearly from the cable profile between beam supports. However, the center of pressure on the concrete will follow the same intrinsic shape as the cable profile. Fig. 2 shows an assumed cable profile and an assumed center of pressure on the concrete for a continuous beam. At the end supports the cable profile and the center of pressure


Fig. 1-Moment Created by Prestressing


Fig. 2-Cable Profile and Center of Pressure on Concrete for End Span
on the concrete coincide. At the interior support, the cable profile and center of pressure on the concrete do not coincide. At the center line of the exterior span, the deviation between the center of pressure on the concrete and the cable profile is half the amount it is at the support.

In continuous prestressed concrete the line of pressure on the concrete is of the utmost importance in the elastic analysis of the beam. The line of pressure must fall within a limiting zone along the beam. The limiting zone is determined from the maximum and minimum moment curves which are created from external loads, cross-sectional properties of the beam, the allowable tension in the concrete used, and the force of the prestressing cable. Lin $^{5}$ discusses the location of the limiting zone in his book. The limiting zone will be briefly discussed here. Fig. 3 shows an elevation of


Fig. 3-Possible Limiting Zone for End Span
the end span of a continuous beam. The vertical scale of the beam is increased for convenience of plotting the limiting zone. The usual beam sign convention is used. A positive moment creates compression stress at the top fiber of the beam. $M_{\max }$
is the largest algebraic moment along the beam. $M_{\text {min }}$ is the smallest algebraic moment along the beam. For clarification, $M_{\max }$ is associated with the maximum positive moment in a beam span and $M_{\text {min }}$ is associated with the support moment for a continuous beam. $F$ is the prestressing force of the cable. When no tension stress is allowed in the concrete, the limiting zone is determined by plotting from the kern lines of the beam. Fig. 3 shows a beam in which no tension stress is allowed in the concrete. The top portion of the limiting zone is determined by the $M_{\max }$ curve. Let $a_{\max }=M_{\max } / \mathrm{F}$. Then $a_{\max }$ is plotted from the top kern line and varies along the beam. When $M_{\text {max }}$ is positive, $a_{\max }$ is plotted below the kern line. When $M_{\text {max }}$ is negative, $a_{\text {max }}$ is plotted above the kern line. Let $a_{\text {min }}=M_{\text {min }} / \mathrm{F}$. When $M_{\text {min }}$ is positive, $a_{\text {min }}$ is plotted below the bottom kern line. When $M_{m i n}$ is negative, $a_{m i n}$ is plotted above the bottom kern line. The zone between the curves obtained by plotting the values of $a_{\operatorname{maz}}$ and $a_{\min }$ is the limiting zone of the beam.

The method used in this discussion is the method of equivalent loads. By this procedure, forces which are created by the action of the prestressing cable pressing against the concrete are determined. These equivalent loads have been determined previously by other writers $^{1,2,4,5}$. For clarification, these loads will also be derived in this paper.

Fixed-end moments due to prestressing will be derived for various cable profiles. These fixed-end moments may be used in the Moment Distribution Method or Slope-Deflection Method to determine support moments in the beam due to the prestressing force. By using these
support moments in the beam, with no live load or dead load considered, the location of a corresponding center of pressure on the concrete at the supports can be determined. The support moment due to prestressing when divided by the prestressing force in the cable locates the center of pressure on the concrete at the supports. Between supports the center of pressure on the concrete varies linearly from the cable profile. Hence, the center of pressure of the concrete from cable alone, can be located throughout the continuous beam.

## CABLE PROFILES STRAIGHT LINE SEGMENTS

Fig. 4 is a portion of a beam in which sharp bends in the cable occur at Section A-A and Section B-B. The cable between these sections is a straight line. The cable makes an angle, $\alpha$, with the horizontal axis of the beam. Assume no friction loss in the cable between the two sections. Also, assume the horizontal component of the cable force is equal to the cable force. The vertical component of the cable force, $F$, at Section $A-A$ and Section $B-B$ is

$$
\begin{equation*}
P_{A}=P_{B}=F \tan \alpha \tag{1}
\end{equation*}
$$

However, since $\tan \alpha=t / k L$, Eq. 1 becomes

$$
\begin{equation*}
P_{A}=P_{B}=\frac{F t}{k L} \tag{2}
\end{equation*}
$$

These are likewise the forces the concrete has to exert on the cable at $A$ and $B$ to keep the cable in the position shown in Fig. 4 when the cable is made horizontal to the left of $A$ and to the right of $B$. Hence Eq. 2 gives the magnitude of the equivalent concentrated load due to this form of prestressing. From Eq. 2 , note that the equivalent concen-


Forces acting on concrete ore shown.
Forces on steel are same in opposite direction.
Fig. 4-Portion of Beam with Straight Cable
trated load is a function only of the force in the cable and the dimensions of the cable profile.
Figs. 5(a) and 5(b) show cable profiles for an end span. The cable profile is composed of straight line segments with sharp bends. The beams are shown with a hinged-end at $A$ and a fixed-end at $B$. The fixedend moment at $B$ will be determined for the cable profile shown in Figs. $5(a)$ or $5(b)$ where $R$ and $S$ are fractional factors applied to the deflection $y$. Two equivalent concentrated loads due to prestressing will occur in the span. Two other concentrated loads due to prestressing will occur at the supports. However, the loads at the supports will not affect the fixed-end moment at $B$.


Fig. 5-Possible Straight Line Segment Cable Profile for End Spans


Fig. 6-Propped Cantilever Beam with Two Concentrated Loads

Fig. 6 shows a beam which is subjected to two concentrated loads. These concentrated loads are assumed positive when acting in a downward direction. The sign convention to be used for the fixed-end moments is the beam bending moment sign convention, that is, a positive moment is a moment which causes compression stress on the top fiber of the beam. By using Eq. 2 and the symbols and dimensions shown in Figs. $5(a)$ and $5(b)$, values for $P_{1}$ and $P_{2}$ shown in Fig. 6 can be obtained.

$$
\begin{gathered}
P_{1}=-F\left[\frac{R y}{a L}+\frac{R y-S y}{b L}\right]= \\
\frac{F y}{L}\left[\frac{R(a+b)-a S}{a b}\right]
\end{gathered}
$$

and

$$
\begin{align*}
P_{2} & =-F\left[\frac{y}{c L}+\frac{S y-R y}{b L}\right]= \\
& -\frac{F y}{L}\left[\frac{b+c(S-R)}{c b}\right] \tag{4}
\end{align*}
$$

A statically determined moment, $m$, is applied at the hinged end $A$ by the cable profiles in Figs. 5(a) and $5(b)$. The value of $m$ is

$$
\begin{equation*}
m=F e_{A} \tag{5}
\end{equation*}
$$

where $F$ is the prestressing force and $e_{A}$ is the distance of the cable from the centroid of the beam at support $A$. When the cable is above the centroid of the beam, $e_{A}$ is positive.

Fig. 7 shows a propped cantilever beam with a concentrated load, $P$, at a distance $x$ from the hinged-end. The equation for the moment, $M_{B}$, can be found in several engineering handbooks.

$$
\begin{equation*}
M_{B}=\frac{P x\left(L^{2}-x^{2}\right)}{2 L^{2}} \tag{6}
\end{equation*}
$$

If $P=P_{1}$ and $x=a L$ are substituted into Eq. 6 and if $P=P_{2}$ and $x=(1-c) L$ are substituted into Eq. 6 and these results are combined


Fig. 7-Propped Cantilever Beam with One Concentrated Load
algebraically into one equation, the following result is obtained

$$
M_{B}=-\frac{P_{1} L a\left(1-a^{2}\right)}{2}-
$$

$$
\begin{equation*}
\frac{P_{2}(1-c) L\left[L^{2}-(1-c)^{2} L^{2}\right]}{2 L^{2}} \tag{7}
\end{equation*}
$$

Substitute the values of $P_{1}$ and $P_{2}$ that are given in Eqs. 3 and 4 into Eq. 7.

$$
\begin{align*}
& M_{B}=\frac{F y}{2 b}\left\{[a(R-\mathrm{S})+b R]\left(1-a^{2}\right)+\right. \\
& {[b+c(\mathrm{~S}-R)](1-c)(2-c)\} } \tag{8}
\end{align*}
$$

Fig. 8 shows a propped cantilever beam with an applied moment, $m$, at the hinged end. The fixed-end moment at $B$ due to the applied moment at $A$ is

$$
\begin{equation*}
M_{B}=-\frac{m}{2}-=\frac{F e_{A}}{2} \tag{9}
\end{equation*}
$$



Fig. 8-Propped Cantilever Beam with Moment Applied at Hinged End
since the applied moment at $A$ is given in Eq. 5.

If Eqs. 8 and 9 are added together algebraically, the fixed-end moments due to prestressing are determined for the cable profiles shown in Figs. $5(a)$ or $5(b)$.

$$
\begin{align*}
M_{B}=\frac{F y}{2 b}\{ & {[a(R-S)+b R]\left(1-a^{2}\right)+} \\
& {[b+c(S-R)](1-c)(2-c)\} } \\
& -\frac{F e_{A}}{2} \tag{10}
\end{align*}
$$

Eq. 10 contains 7 independent variables and one dependent variable. The independent variables are $F, y, e_{A}, a, b, R$ and $S$. Since $a+b+c=1, c$ is not a variable after $a$ and $b$ are assigned values. The dependent variable of Eq. 10 is $M_{B}$.

One design chart that included all the variables of Eq. 10 would be very difficult to construct. However, three separate design tables are presented which will allow rapid calculation of $M_{B}$. Eq. 9 was added algebraically with Eq. 8 to obtain Eq. 10. Eq. 9 is a very simple equation. Therefore, it will not be included in the design tables. Eq. 9 reveals that the fixed-end moment, $M_{B}$, is equal to one-half the magnitude and opposite in sign to the applied moment created by the perestressing at support $A$.

The three design tables will be determined from Eq. 8.

Assume $R=S=0$ and substitute February 1966
these values in Eq. 8. The result is

$$
\begin{equation*}
M_{B}=\frac{F y(1-c)(2-c)}{2} \tag{11}
\end{equation*}
$$

Divide each side of Eq. 11 by Fy.

$$
\begin{equation*}
\frac{M_{B}}{F \underline{v}}=\frac{(1-c)(2-c)}{2} \tag{12}
\end{equation*}
$$

The following equation is obtained from differential calculus:

$$
\begin{equation*}
\Delta M_{B}=\left(\frac{\partial M_{B}}{\partial R}\right) \Delta R \tag{13}
\end{equation*}
$$

Where $\Delta M_{B}$ is the change in the fixed-end moment $M_{B}, \Delta R$ is the change in the variable $R$, and $\frac{\partial M_{B}}{\partial R}$ is the partial derivative of $M_{B}$ with respect to $R$. Take the partial derivative of Eq. 8,

$$
\begin{array}{r}
\frac{\partial M_{B}}{\partial R}=\frac{F y}{2 b}\left[(a+b)\left(1-a^{2}\right)-\right. \\
c(1-c)(2-c)] \tag{14}
\end{array}
$$

Substitute Eq. 14 into Eq. 13,

$$
\begin{align*}
\Delta M_{B}= & \frac{F y}{2 k}\left[(a+b)\left(1-a^{2}\right)-\right. \\
& c(1-c)(2-c)](\Delta R) \tag{15}
\end{align*}
$$

The following equation also comes from calculus:

$$
\begin{equation*}
\Delta M_{B}=\left(\frac{\partial M_{B}}{\partial S}\right)(\Delta S) \tag{16}
\end{equation*}
$$

Applying calculus to Eq. 8,

$$
\begin{aligned}
\frac{\partial M_{B}}{\partial S}= & \frac{F y}{2 b}\left[-a\left(1-a^{2}\right)+\right. \\
& c(1-c)(2-c)] \Delta S(17)
\end{aligned}
$$

Substitute Eq. 17 into Eq. 16,

$$
\begin{align*}
\Delta M_{B}=\frac{F y}{2 b}[ & -a\left(1-a^{2}\right)+ \\
& c(1-\mathrm{c})(2-c)] \Delta S) \tag{18}
\end{align*}
$$

Let $\Delta R=R$ in Eq. 15 and divide
both sides of Eq. 18 by FyS.

$$
\begin{align*}
\frac{\Delta M_{B}}{F y R}= & \frac{(a+b)\left(1-a^{2}\right)}{2 b}+ \\
& \frac{c(1-c)(2-c)}{2 b} \tag{19}
\end{align*}
$$

Let $\Delta S=S$ in Eq. 18 and divide both sides of Eq. 18 by FyS.

$$
\begin{align*}
\frac{\Delta M_{B}}{F y S}= & \frac{-a\left(1-a^{2}\right)}{2 b}+ \\
& \frac{c(1-c)(2-c)}{2 b} \tag{20}
\end{align*}
$$

The change of the fixed-end moment due to prestressing which is caused by the variables $R$ and $S$ can be calculated by Eqs. 19 and 20. Eq. 12 can be used to calculate $M_{B}$ when $R=S=0$. Hence, if Eqs. 12, 19 and 20 are added together algebraically, the fixed-end moment due to prestressing can be calculated.

The solutions of Eqs. 12, 19 and 20 are given in Tables 1, 2 and 3 respectively. The increments of the tabulated values are such that straight line interpolation may be used while creating a maximum error of 10 in the fifth decimal place.

Fig. 9(a) shows a propped cantilever beam with a cable profile. The magnitude of the prestressing force is assumed constant along the en-

(a)

(b)

Fig. 9-Straight Line Segment Profile for End Span
tire span. The design tables, Tables 1,2 and 3 , will be used to calculate the fixed-end moment. Eq. 9 also will be used to calculate the fixedend moment for $F=100 \mathrm{kips}$.

$$
\begin{aligned}
M_{B 1}= & -\frac{F e_{A}}{2}=-\frac{100(0.25)}{2}= \\
& -12.5 \text { kip-ft. }
\end{aligned}
$$

When $a=0.2$ and $c=0.4$, the fixedend moment due to Table 1 is

$$
\begin{aligned}
M_{B 2} & =+0.48 F y \\
& =+0.48(100)(1.5) \\
& =+72 \mathrm{kip}-\mathrm{ft} .
\end{aligned}
$$

When $a=0.2, c=0.4$ and $R=1 / 3$,

Table 1-Values of
$\left.\frac{M_{B}}{F y}\right|_{1}=\frac{(1-c)(2-c)}{2}$

| $c$ | Value | c | Value | c | Value | $c$ | Value |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0.00 | 1.0000 | 0.20 | 0.7200 | 0.40 | 0.4800 | 0.60 | 0.2800 |
| 0.02 | 0.9702 | 0.22 | 0.6942 | 0.42 | 0.4582 | 0.62 | 0.2622 |
| 0.04 | 0.9408 | 0.24 | 0.6688 | 0.44 | 0.4368 | 0.64 | 0.2448 |
| 0.06 | 0.9118 | 0.26 | 0.6438 | 0.46 | 0.4158 | 0.66 | 0.2278 |
| 0.08 | 0.8832 | 0.28 | 0.6192 | 0.48 | 0.3952 | 0.68 | 0.2112 |
| 0.10 | 0.8550 | 0.30 | 0.5950 | 0.50 | 0.3750 | 0.70 | 0.1950 |
| 0.12 | 0.8272 | 0.32 | 0.5712 | 0.52 | 0.3552 | 0.72 | 0.1792 |
| 0.14 | 0.7998 | 0.34 | 0.5478 | 0.54 | 0.3358 | 0.74 | 0.1638 |
| 0.16 | 0.7728 | 0.36 | 0.5248 | 0.56 | 0.3168 | 0.76 | 0.1488 |
| 0.18 | 0.7462 | 0.38 | 0.5022 | 0.58 | 0.2982 | 0.78 | 0.1342 |



| $\leq$ |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| c | 0.20 |  |  | , | 0.28 | 0.30 | 32 |  | , | 0.38 | . 4 |
| 0.00 | 6000 | 6100 | 6200 | 6300 | 6400 | 6500 | 6600 | 6700 | 800 | 6900 | 7000 |
| 0.02 | 5782 | 5880 | 5978 | 5076 | 6174 | 5272 | 6370 | 6468 | 6565 | 6664 | 5762 |
| 0.04 | 5568 | 5664 | 5760 | 5856 | 5952 | 6048 | 6144 | 6240 | 6336 | 6432 | 6528 |
| 0.06 | 5358 | 5452 | 5546 | 5640 | 5734 | 5828 | 5922 | 6016 | 6110 | 6204 | 6298 |
| 0.08 | 5152 | 5244 | 5336 | 5428 | 5520 | 5612 | 5704 | 5796 | 5888 | 5980 | 6072 |
| 0.10 | 4950 | 5040 | 5130 | 5220 | 5310 | 5400 | 5490 | 5580 | 5670 | 5760 | 5850 |
| 0.12 | 4752 | 4840 | 4928 | 5016 | 5104 | 5192 | 5280 | 5368 | 5456 | 5544 | 5632 |
| 0.14 | 4558 | 4644 | 4730 | 4816 | 4902 | 4988 | 5074 | 5160 | 5246 | 5332 | 5418 |
| 0.16 | 4368 | 4452 | 4536 | 4620 | 4704 | 4788 | 4872 | 4956 | 5040 | 5124 | 5208 |
| 0.18 | 4182 | 4264 | 4346 | 4428 | 4510 | 4592 | 4674 | 4756 | 4838 | 4920 | 5002 |
| 0.20 | 4000 | 4080 | 4160 | 4240 | 4320 | 4400 | 4480 | 4560 | 4640 | 4720 | 800 |
| 0.22 | 3822 | 3900 | 3978 | 4056 | 4134 | 4212 | 1.290 | 4368 | 4446 | 4524 | 4602 |
| 0.24 | 3648 | 3724 | 3800 | 3876 | 3952 | 4028 | 4104 | 4180 | 4256 | 4332 | 4.408 |
| 0.26 | 3478 | 3552 | 3626 | 3700 | 3774 | 3848 | 3922 | 3996 | 4070 | 4144 | 218 |
| 0.28 | 3312 | 3384 | 34.56 | 3528 | 3600 | 3672 | 374.4 | 3816 | 3888 | 3960 | 4032 |
| 0.30 | 3150 | 3220 | 3290 | 3360 | 3430 | 3500 | 3570 | 3640 | 3710 | 3780 | 20 |
| 0.32 | 2992 | 3050 | 3128 | 3195 | 3264 | 3332 | 3400 | 3468 | 3536 | 3604 | 672 |
| 0.34 | 2838 | 2904 | 2970 | 3036 | 3102 | 3168 | 3234 | 3300 | 3366 | 3432 | 3498 |
| 0.36 | 2688 | 2752 | 2816 | 2880 | 2944 | 3008 | 3072 | 3136 | 3200 | 3264 | 3328 |
| 0.38 | 2542 | 2604 | 2666 | 2728 | 2790 | 2852 | 2914 | 2976 | 3038 | 3100 | 3162 |
| 0.40 | 2400 | 2460 | 2520 | 2580 | 2640 | 2700 | 2760 | 2820 | 2880 | 2940 | 3000 |
| 0.42 | 2262 | 2320 | 2378 | 2436 | 2494 | 2552 | 2610 | 2668 | 2726 | 2786 | 2842 |
| 0.44 | 2128 | 2184 | 2240 | 2296 | 2352 | 2408 | 2464 | 2520 | 2576 | 2632 | 2688 |
| 0.46 | 1998 | 2052 | 2106 | 2160 | 2214 | 2268 | 2322 | 2376 | 2430 | 2484 | 2538 |
| 0.48 | 1872 | 192.4 | 1976 | 2028 | 2080 | 2132 | 2184 | 2236 | 2288 | 2340 | 2392 |
| 0.50 | 1750 | 1800 | 1850 | 1900 | 1950 | 2000 | 2050 | 2100 | 2150 | 2200 |  |
| 0.52 | 1632 | 1680 | 1728 | 1776 | 1824 | 1872 | 1920 | 1968 | 2016 | 2064 | 2112 |
| 0.54 | 1518 | 1564 | 1610 | 1656 | 1702 | 1748 | 1794 | 1840 | 1886 | 1932 | 1978 |
| 0.56 | 1408 | 14.52 | 1496 | 1540 | 1584 | 1628 | 1672 | 1716 | 1760 | 1804 | 1848 |
| 0.58 | 1302 | 1344 | 1386 | 1428 | 1470 | 1512 | 1554 | 1596 | 163 ! | 1680 |  |
| 0.60 | 1200 | 1240 | 1280 | 1320 | 1360 | 14,00 | 14.40 | 1480 | 1520 |  |  |
| 0.62 | 1102 | 1140 | 1178 | 1216 | 1254 | 1292 | 1330 | 1368 |  |  |  |
| 0.64 | 1008 | 1044 | 1020 | 1116 | 1152 | 1188 | 1224 |  |  |  |  |
| 0.66 | 0918 | 0952 | 0986 | 1020 | 1054 | 1088 |  |  |  |  |  |
| 0.68 | 0832 | 0864 | 0896 | 0928 | 0960 |  |  |  |  |  |  |
| 0.70 | 0750 | 0780 | 0810 | 0540 |  |  |  |  |  |  |  |
| 0.72 | 0672 | 0700 | 0728 |  |  |  |  |  |  |  |  |
| 0.74 | 0598 | 0624 |  |  |  |  |  |  |  |  |  |
| 0.76 | 0528 |  |  |  |  |  |  |  |  |  |  |

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| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| C | 0.20 | 0.22 | 0.24 | 0.26 | 0.28 | 0.30 | 0.32 | O | 0.36 | 0.38 | 0.40 | 0.42 | 0.44 | 0.46 | 0.48 | 0.50 | 0.52 | 0.54 | 0.56 | 0.58 | 0.60 |
| 0.00 | -1200 | -1342 | -1488 | $-1638$ | -1792 | -1950 | -2112 | -2278 | $-2448$ | -2622 | -2800 | -2982 | -3168 | -3358 | -3552 | -3750 | -3952 | $-4158$ | $-4368$ | -4582 | $-4800$ |
| 0.02 | -0982 | -1122 | -1266 | -1414 | -1566 | -1722 | -1882 | -2046 | $-2214$ | -2386 | -2562 | -2742 | -2926 | -3114 | -3306 | -2502 | -3702 | -3906 | -4114 | -4326 | $-4542$ |
| 0.04 | -0768 | -0906 | -1048 | -1194 | -1344 | -1498 | -1656 | -1818 | -1984 | -2154 | -2328 | -2506 | -2688 | $-2874$ | -3064 | -3258 | -3456 | -3658 | -3864 | -4074 | -4288 |
| 0.06 | -0558 | -0694 | -0834 | -0978 | -1126 | -1278 | -1434 | -1594 | -1758 | -1926 | -2098 | -22.74 | -2454 | -2638 | -2826 | -2018 | -3214 | -3414 | -3618 | -3826 | -4,038 |
| 0.08 | -0352 | -1486 | -0624 | -0766 | -0912 | -1062 | -1216 | -1374 | -1536 | -1702 | -1872 | -2046 | -2224 | -2406 | -2592 | -2782 | -2976 | -3174 | -3376 | -3582 | -3792 |
| 0.10 | -0150 | -0292 | -0418 | -0558 | -0702 | -0850 | -1002 | -1158 | -1318 | $-1482$ | -1650 | -1822 | -1998 | -2178 | -2362 | -2550 | -2742 | -2938 | -3138 | -3342 | -3550 |
| 0.12 | 0048 | -0082 | -0216 | -0354 | -0496 | -0642 | -0792 | -0946 | -1104 | -1266 | -1432 | -1602 | -1776 | -1954 | -2136 | -2322 | -2512 | -2706 | -2904 | -3106 | -3312 |
| 0.14 | 0242 | 0114 | -0018 | -0154 | -0294 | -0438 | -0586 | -0738 | -0894 | -1054 | -1218 | -1386 | -1558 | -1734 | -1914 | -2098 | -2286 | -2478 | -2674 | -2874 | -3078 |
| 0.16 | 0432 | 0306 | 0176 | 0042 | -0096 | -0238 | -0384 | -0534 | -0688 | -0846 | -1008 | -1174 | -1344 | -1518 | -1696 | -1878 | -2064 | -2252 | -24.48 | -2646 | -2848 |
| 0.18 | 0618 | 0494 | 0366 | 0234 | 0098 | -0042 | -0186 | -0334 | -0486 | -0642 | -0802 | -0966 | -1134 | -1306 | -1483 -1272 | -1662 -1450 | -1846 -1632 | -2034 -1818 | -2226 -2008 | -2422 -2202 | -2622 -2400 |
| 0.20 | 0800 | 0678 | 0552 | 0422 | 0288 | 0150 | 0008 | -0138 | -0288 | -0442 | -0600 | -0762 | -0938 | -1098 | -1272 | -1450 | -1632 | -1818 | -2008 | -2202 | -2400 |
| 0.22 | 0978 | 0858 | 0734 | 0606 | 0474 | 0338 | 0198 | 0054 | -0094 | -0246 | -0402 | -0562 | -0726 | -0894 | -1066 | -1242 -1038 | -1422 | -1606 | -1794 -1584 | -1986 | -2182 -1968 |
| 0.24 | 1152 | 1034 | 0912 | 0786 | 0656 | 0522 | 0384 | 0242 | 0096 | -0054 | -0208 | -0366 -0174 | -0528 -0334 | -0694 | -0864 | -1038 | -1216 | -1398 | -1584 | -1754 | -1968 |
| 0.26 | 1322 | 1206 | 1086 | 0962 | 0834 | 0702 | 0566 | 0426 | 0282 | 0134 | -0018 | -0174 | -0334 -014 | -0498 -0306 | -0666 | -0838 | -1014 | -1194 | -1378 | -1566 | $-150$ |
| 0.28 | 14.88 | 1374 | 1256 | 1134 | 1008 | 0878 | 0744 | 0606 | 0464 | 0318 | 0168 | 0014 | -0144 | -0306 | -0472 -0782 | -0642 -0450 | -0816 | -0994 | -1176 | -1362 | $-1552$ |
| 0.30 | 1650 | 1538 | 1422 | 1302 | 1178 | 1050 | 0918 | 0782 | 0642 | 0498 | 0350 | 0198 | 0042 | -0118 | -0722 | -0450 | -0622 | -0798 | 8 | -1162 |  |
| 0.32 | 1808 | 1698 | 1584 | 1466 | 1344 | 1218 | 1088 | 0954 | 0816 | 0674 | 0528 | 0378 | 0224 | 0066 | -0096 | -0262 | -0432 | -0606 | -0784 | -0966 | -1152 |
| 0.34 | 1962 | 1854 | 1742 | 1626 | 1506 | 1382 | 1254 | 1122 | 0986 | 0846 | 0702 | 0554 | 0402 | 0246 | 0086 | -0078 | -0246 | -0418 | -0594 | -0774 | $-0958$ |
| 0.36 | 2112 | 2006 | 1896 | 1782 | 1664 | 1542 | 1416 | 1286 | 1152 | 1014 | 0872 | 0726 | 0576 | 04,22 | 0264 | 0102 | -0064 | -0234 | -0408 |  | -0768 |
| 0.38 | 2258 | 2.154 | 2046 | 1934 | 1818 | 1698 | 1574 | 14.46 | 1314 | 1178 | 1038 | 0894 | 0746 | 0594 | 0438 | 0278 | 0114 | -0054 | -0226 | -0402 |  |
| 0.40 | 2400 | 2298 | 2192 | 2082 | 1968 | 1850 | 1728 | 1602 | 1472 | 1.38 | 1200 | 1058 | 0912 | 0762 | 0608 | 0450 | 0288 | 0122 | -0048 |  |  |
| 0.42 | 2538 | 2438 | 2334 | 2226 | 2114 | 1998 | 1878 | 1754 | 1626 | 1494 | 1358 | 1218 | 1074 | 0926 | 0774 | 0618 | 0458 | 0294 |  |  |  |
| 0.44 | 2672 | 2574 | 2472 | 2366 | 2256 | 2142 | 2024 | 1902 | 1776 | 1646 | 1512 | 1374 | 1232 | 1086 | 0936 | 0782 | 0624 |  |  |  |  |
| 0.46 | 2802 | 2706 | 2606 | 2502 | 2394 | 2282 | 2166 | 2046 | 1922 | 1794 | 1662 | 1526 | 1386 | 1242 | 1094 | 0942 |  |  |  |  |  |
| 0.48 | 2928 | 2 2834 | 2736 | 2634 | 2528 | 2418 | 2304 | 2186 | 2064 | 1938 | 1808 | 1674 | 1536 | 1394 | 1248 |  |  |  |  |  |  |
| 0.50 | 3050 | 2958 | 2862 | 2762 | 2658 | 2550 | 2438 | 2322 | 2202 | 2078 | 1950 | 1818 | 1682 | 1542 |  |  |  |  |  |  |  |
| 0.52 | 3168 | 3078 | 2984 | 2886 | 2784 | 2678 | 2568 | 2454 | 2336 | 2214 | 2083 | 1958 | 1824 |  |  |  |  |  |  |  |  |
| 0.54 | 3282 | 3194 | 3102 | 3006 | 2906 | 2802 | 2694 | 2582 | 2.466 | 2346 | 2222 | 2094 |  |  |  |  |  |  |  |  |  |
| 0.56 | 3392 | 3306 | 3216 | 3122 | 3024 | 2922 | 2816 | 2706 | 2592 | 2474 | 2352 |  |  |  |  |  |  |  |  |  |  |
| 0.58 | 3498 | 3414 | 3326 | 3234 | 3138 | 3038 | 2934 | 2826 | 2714 | 2598 |  |  |  |  |  |  |  |  | by |  |  |
| 0.60 | 3600 | 3518 | 3432 | 3342 | 3248 | 3150 | 3048 | 2942 | 2832 |  |  |  | ote: | ave | 迷 | F | be |  | by |  |  |
| 0.62 | 3698 | 3618 | 3534 | 3446 | 3354 | 3258 | 3158 | 3054 |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 0.64 | 3792 | 3714 | 3632 | 3546 | 3456 | 3362 | 3264 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 0.66 | 3882 | 3806 | 3726 | 3642 | 3554 | 3462 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 0.68 | 3968 | 3894 | 3816 | 3734 | 3648 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 0.70 | 4050 | 3978 | 3902 | 3822 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 0.72 | 4128 | 4058 | 3984 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 0.74 | 4202 | 4134 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 0.76 | 4272 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |

the fixed-end moment due to Table 2 is

$$
\begin{aligned}
M_{B 3} & =+0.24 F y R \\
& =+0.24(100)(1.5)(1 / 3) \\
& =+12 \mathrm{kip}-\mathrm{ft} .
\end{aligned}
$$

For $a=0.2, c=0.4$ and $S=\%$, the fixed-end moment given by Table 3 is

$$
\begin{aligned}
M_{B 4} & =+0.24 \mathrm{FyS} \\
& =+0.24(100)(1.5)(\% / 5) \\
& =+24 \mathrm{kip}-\mathrm{ft} .
\end{aligned}
$$

Hence, the fixed-end moment for the beam shown in Fig. $9(a)$ is obtained by the following algebraic summation.

$$
\begin{aligned}
M_{B}= & -12.5+72+12+24= \\
& +95.5 \mathrm{kip} \text {-ft. }
\end{aligned}
$$

If Fig. $9(a)$ is one span of a twospan continuous beam with equal spans, a concordant cable can be obtained in the following manner. Assume the cable profile can be raised or lowered. Also assume that the cable must keep the same deflected shape as in $9(a)$. Thus, if only $e_{A}$ may be varied and $R$ and $S$ must be retained, the values from the tables remain the same as in the previous calculations. The algebraic summation from these three charts is $M_{B}=72+24+12=108$ kip-ft. The fixed-end moment when $e_{A}$ is a variable is

$$
\begin{align*}
M_{B}= & 108-\frac{F e_{A}}{2} \\
& 108-50 e_{A} \tag{20a}
\end{align*}
$$

For the cable to be concordant, the fixed-end moment must be equal to the force in the cable times the distance of the cable from the centroid of the beam. Hence,

$$
\begin{align*}
M_{B} & =+F\left(e_{A}-1.00+1.50\right) \\
& =+100\left(e_{A}+0.50\right) \tag{21}
\end{align*}
$$

Set Eq. $20 a$ equal to Eq. 21 and solve for $e_{A}$.

$$
\begin{gathered}
100\left(e_{A}+0.50\right)=108-50 e_{A} \\
e_{A}=\frac{+58}{150}=0.387 \mathrm{ft} .
\end{gathered}
$$

Hence, Fig $9(b)$ shows a cable profile which is a concordant cable for a two span continuous beam of equal spans. The beam and cable profile is symmetrical about support $B$.

Fig. 10 shows a symmetrical cable profile for an interior span composed of straight line segments with sharp bends.
(A)


Fig. 10-Symmetrical Straight Line Segment Profile for Interior Span

In a three-span continuous unit, the moment curves will usually be symmetrical about the center-line of the middle span. In a four-span continuous unit, the point of maximum positive moment will not occur exactly at the centerline of the interior span. However, the maximum positive moment will occur closer to the centerline of the interior span than it will in the exterior span. Only a cable profile which is symmetrical about the centerline of span will be considered.
Two equivalent concentrated loads due to prestressing will be created by the cable profile shown in Fig. 10. These loads will be located at a distance $a L$ from each support. Fig. $11(a)$ shows equal positive, i.e., downward, loads acting on the beam. Applying the dimensions shown in Fig. 10 with Eq. 2, the magnitude of the equivalent concen-

(a) Fixed-End Beam with Symmetrical Concentrated Loads

(b) Simple Beam Moment Diagram

(c) Moment Diagram Due to Fixed-End Moments

Fig. 11-Fixed-End Beam
trated loads due to prestressing is

$$
\begin{equation*}
P=-\frac{F y}{a L} \tag{22}
\end{equation*}
$$

Fig. $11(b)$ shows the moment diagram for a simple beam with two equal concentrated loads at a distance $a L$ from each end. Fig. 11(c) shows the moment diagram which is created by equal end moments. When a beam is fixed at each end and subjected to the symmetrical loads shown in Fig. 11(a), the total moment diagram for the beam can be obtained by combining the moment diagrams shown in Figs. $11(b)$ and $11(c)$ to make the total area
equal to zero (zero angle change from $A$ to $B$ ). The fixed-end moments are then

$$
\begin{equation*}
M_{A}=M_{B}=-P a L(1-a) \tag{23}
\end{equation*}
$$

Substitute Eq. 22 into Eq. 23,

$$
\begin{equation*}
M_{A}=M_{B}=+F y(1-a) \tag{24}
\end{equation*}
$$

Eq. 24 gives the fixed-end moments due to prestressing for the cable profile shown in Fig. 10.

## CABLE PROFILE PARABOLIC CURVES

Fig. 12 shows a portion of a beam with a constant cross-section that has a parabolic shaped cable profile. Section $B-B$ is passed through the beam where the slope of the parabola is horizontal. Section A-A is a section which is a distance $k L$ from Section B-B. The equation of the parabola with respect to its horizontal tangent at Section $B-B$ is $z=c x^{2}$.


Fig. 12-Equivalent Uniform Load Due to Parabolic Cable

Since the cable profile is parabolic, the moment curve induced by the prestressing force is parabolic. From elementary structural theory, if a moment curve is a parabola, the load condition which produces it is a uniform load. Again it is assumed that the loss due to friction is negligible and the horizontal component of the cable force taken is equal to the total force on the cable. Section $B-B$ is a point of zero shear because the slope of the moment curve, like
the cable, is horizontal when the beam is considered under the action of the prestressing force alone. Let $w_{F}$ be the equivalent uniform load due to prestressing. $F$ is the force in the cable, $t$ is the vertical rise of the parabola, and $k L$ is the horizontal length of the parabola being considered. Take moments at Section A-A at the point of the cable and set this moment summation equal to zero.

$$
\begin{equation*}
\Sigma M=F t-\frac{w_{F} k^{2} L^{2}}{2}=0 \tag{25}
\end{equation*}
$$

Solve Eq. 25 for $w_{P}$.

$$
\begin{equation*}
w_{F}=\frac{2 F t}{k^{2} L^{2}} \tag{26}
\end{equation*}
$$

Fig. 13 shows an end span with a cable profile composed of three different parabolas. The cable pro-


Fig. 13-Possible Parabolic Profile for End Span
file is continuous and the parabolas have a common tangent at their points of intersection at $G$ and $H$. The equations for these curves are

$$
\begin{align*}
& z=\frac{x^{2} R y}{a^{2} L^{2}}  \tag{27}\\
& z=\frac{x^{2} m}{b^{2} L^{2}} \tag{28}
\end{align*}
$$

and

$$
\begin{equation*}
z=\frac{x^{2} n}{c^{2} L^{2}} \tag{29}
\end{equation*}
$$

Eq. 27 and 28 have horizontal tangents at their origins, i.e., $x=0$, which is the common point between February 1966
the two curves. Eq. 29 has a horizontal tangent at the fixed end of the beam. For Eq. 28 and 29 to have a common tangent at their intersection, the derivatives of each curve must be equal at point $H$. The derivative of Eq. 28 at $x=b L$ is

$$
\begin{equation*}
\frac{d z}{d x}=\frac{2 m x}{b^{2} L^{2}}=\frac{2 m}{b L} \tag{30}
\end{equation*}
$$

The derivative of Eq. 29 at $x=c L$ is

$$
\begin{equation*}
\frac{d z}{d x}=\frac{2 n x}{c^{2} L^{2}}=\frac{2 n}{c L} \tag{31}
\end{equation*}
$$

Equate Eqs. 30 and 31 and combine terms,

$$
\begin{equation*}
\frac{m}{n}=\frac{b}{c} \tag{32}
\end{equation*}
$$

From Fig. 13, it can be seen that $a+b+c=1$ and $m+n=y$. Thus with these two equations and Eq. $32, m$ and $n$ can be expressed in terms of $c, b, a$ and $y$.

$$
\begin{gather*}
n=y-m=y-\frac{n b}{c}= \\
\frac{y c}{a+b}=\frac{y c}{(1-c)} \tag{33}
\end{gather*}
$$

and

$$
\begin{gather*}
m=y-n=y-\frac{c m}{b}= \\
\frac{y b}{c+b}=\frac{y b}{(1-a)} \tag{34}
\end{gather*}
$$

Let $w_{a}$ be the equivalent uniform load over the $a L$ portion of the span, $w_{b}$ the equivalent uniform load over the $b L$ portion of the span, and $w_{c}$ the equivalent uniform load over the $c L$ portion of the span. The uniform loads are considered positive when acting in a downward direction. Using the above load notation and the dimensions shown in Fig. 13, the Eqs. 26, 33 and 34, the equations for the equivalent uniform
loads are

$$
\begin{align*}
& w_{a}=-\frac{2 F R y}{a^{2} L^{2}}  \tag{35}\\
& w_{b}=-\frac{2 F y}{b(1-a) L^{2}} \tag{36}
\end{align*}
$$

and

$$
\begin{equation*}
w_{c}=+\frac{2 F y}{c(1-a) L^{2}} \tag{37}
\end{equation*}
$$

Equations for the fixed-end moments of a propped cantilever beam with a uniform load over portions of the span will now be derived. Fig. 7 shows a propped cantilever beam with a concentrated load at $x$ distance from the hinged-end. The equation for the fixed-end moment for the beam in Fig. 7 is

$$
\begin{equation*}
M_{B}=-\frac{P x\left(L^{2}-x^{2}\right)}{2 L^{2}} \tag{6}
\end{equation*}
$$

Fig. 14 shows a method of expressing a uniform load as an infinite number of concentrated loads along a beam. The magnitude of these con-


Fig. 14-Method of Expressing Uniform Load as Infinite Number of Concentrated Loads
centrated loads is $P=w d x$, where $w$ is the uniform load on the beam and $d x$ is an infinitely small length of the beam. Substitute this value
of the concentrated load into Eq. 6.

$$
\begin{equation*}
M_{n}=-\frac{(w d x) x\left(L^{2}-x^{2}\right)}{2 L^{2}} \tag{38}
\end{equation*}
$$

Eq. 38 is the equation for the fixedend moment of a propped cantilever beam due to a uniform load over an infinitely small length of beam at a distance $x$ from the hinged-end. The equation for fixed-end moment due to a uniform load over a portion of the span can be obtained by integrating Eq. 38 between the limits of the load.

In Eq. 38, let $w=w_{a}$ and integrate between the limits of $x=0$ and $x=a L$.

$$
\begin{align*}
M_{B} & =\int_{0}^{a L} \frac{w_{a} x\left(L^{2}-x^{2}\right) d x}{2 L^{2}} \\
& =-\frac{w_{a}}{2 L^{2}}\left[\frac{L^{2} x^{2}}{2}-\frac{x^{4}}{4}\right]_{0}^{a L} \\
& =-\frac{w_{a} L^{2} a^{2}}{8}\left(2-a^{2}\right) \tag{39}
\end{align*}
$$

Eq. 39 is the equation for the fixedend moment at $B$ for the beam in Fig. 15(a).
Let $w=w_{0}$ in Eq. 38. Integrate Eq. 38 between the limits of $x=a L$ and $x=(a+b) L$.

$$
\begin{align*}
M_{B}= & \int_{a L}^{(a+b) L} \frac{-w_{b} x\left(L^{2}-x^{2}\right) d x}{2 L^{2}}= \\
& \frac{w_{b}}{2 L^{2}}\left[\frac{L^{2} x^{2}}{2}-\frac{x^{4}}{4}\right]_{a L}^{(a+s) L}= \\
- & \frac{w_{b}}{2 L^{2}}\left[2 L^{4}(a+b)^{2}-\right. \\
& \left.(a+b)^{4} L^{4}-2 a^{2} L^{4}+a^{4} L^{4}\right]= \\
- & \frac{w_{b} L^{2}}{8}\left[(a+b)^{2}\left[2-(a+b)^{2}\right]-\right. \\
& \left.a^{2}\left(2-a^{2}\right)\right] \tag{40}
\end{align*}
$$

Eq. 40 is the fixed-end moment at $B$ for the beam in Fig. 15(b).

The equation of the fixed-end moment for the beam in Fig. 15(c)


Fig. 15-Three Loading Conditions for Uniform Load Applied to a Propped Cantilever Beam
will now be derived. Let $w=w_{c}$ in Eq. 38. Integrate Eq. 38 between the the limits of $x=(1-c) L$ and $x=L$.

$$
\begin{align*}
M_{B} & =\int_{a-o L}^{L}-\frac{w_{c} x\left(L^{2}-x^{2}\right) d x}{2 L^{2}} \\
& =\frac{w_{c}}{2 L^{2}}\left[\frac{L^{2} x^{2}}{2}-\frac{x^{4}}{4}\right]_{(1-c) t}^{L} \\
& =-\frac{w_{c} L^{2}}{8}\left[1-(1-c)^{2}\right]^{2} \\
& =-\frac{w_{c} L^{2} c^{2}(2-c)^{2}}{8} \tag{41}
\end{align*}
$$

The equation for the fixed-end moment of a propped cantilever beam due to an applied moment at the hinged-end has appeared previously. This equation was

$$
\begin{equation*}
M_{B}=-\frac{m}{2}=-\frac{F e_{A}}{2} \tag{9}
\end{equation*}
$$

Add algebraically Eqs. 39, 40, 41 and 9 .

$$
\begin{array}{r}
M_{B}= \\
\quad \frac{w_{a} L^{2} a^{2}\left(2-a^{2}\right)}{8}-\frac{w_{b} L^{2}}{8} \\
{\left[(a+b)^{2}\left[2-(a+b)^{2}\right]-a^{2}\left(2-a^{2}\right)\right]} \\
\quad-\frac{w_{c} L^{2} c^{2}(2-c)^{2}}{8}-\frac{m}{2}(42)
\end{array}
$$

Substitute Eqs. 35, 36 and 37 into Eq. 42.

$$
\begin{align*}
M_{B}=\frac{F y}{2}[ & \left(2-a^{2}\right)\left(R-\frac{a^{2}}{b(1-a)}\right)+ \\
& \frac{(a+b)^{2}\left[2-(a+b)^{2}\right]}{b(1-a)}- \\
& \left.\frac{c(2-c)^{2}}{(1-a)}\right]-\frac{F e_{A}}{2} \tag{43}
\end{align*}
$$

If the cable location at $A$ coincides with the centroid of the beam, $e_{A}=0$. If $e_{A}=0$, Eq. 43 reduces to

$$
\begin{align*}
& M_{n}=+\frac{F y}{4}\left[\left(2-a^{2}\right)\left(R-\frac{a^{2}}{b(1-a)}\right)\right. \\
&+\frac{(a+b)^{2}\left[2-(a+b)^{2}\right]}{b(1-a)}- \\
&\left.\frac{c(2-c)^{2}}{(1-a)}\right] \tag{44}
\end{align*}
$$

Substitute $a+b=1-c$ into the middle term of Eq. 44.

$$
\begin{array}{r}
M_{B}=\frac{F y}{4}\left[\left(2-a^{2}\right)\left(R-\frac{a^{2}}{b(1-a)}\right)+\right. \\
\frac{(1-c)^{2}\left(1+2 c-c^{2}\right)}{b(1-a)}- \\
\left.\frac{c(2-c)^{2}}{(1-a)}\right] \tag{45}
\end{array}
$$

Eq. 44 or Eq. 45 will be used to develop design tables that will enable a rapid calculation of $M_{B}$. A correction can easily be made if $e_{A}$ does not equal zero.
Design tables for Eq. 44 or Eq. 45 are Tables 4 and 5. In calculus it was shown that

$$
\begin{equation*}
\Delta M_{B}=\left(\frac{\partial M_{B}}{\partial R}\right) \Delta R \tag{46}
\end{equation*}
$$

In Eq. 46, $\Delta M_{B}$ is the change in the fixed-end moment at $B, \Delta R$ is the change in the variable $R$, and $\frac{\partial M_{B}}{\partial R}$ is the partial derivative of $M_{B}$ with respect to $R$. The partial derivative of Eq. 45 is

$$
\begin{equation*}
\frac{\partial M_{B}}{\partial R}=\frac{F y}{4}\left(2-a^{2}\right) \tag{47}
\end{equation*}
$$

Substitute Eq. 47 into Eq. 46,

$$
\begin{equation*}
\Delta M_{B}=\frac{+F y}{4}\left(2-a^{2}\right)(\Delta R) \tag{48}
\end{equation*}
$$

If $F, y$ and $a$ are constant in Eq. 48, then the change in $M_{B}$ is proportional to the change in $R$. Eq. 48 reveals that if the value of $a$ is constant, $R$ produces a linear effect on the fixed-end moment at $B$.

For Tables 4 and 5, straight line interpolation may be used with the

Table 4
Solution of $\frac{\Delta M_{B}}{F y(\Delta R)}=\frac{\left(2-a^{2}\right)}{4}$

|  | $\frac{2-a^{2}}{4}$ |
| :---: | :---: |
| $a$ | 0.469375 |
| 0.35 | 0.4697600 |
| 0.36 | 0.4675775 |
| 0.37 | 0.4653900 |
| 0.38 | 0.46390 |
| 0.39 | 0.461975 |
| 0.40 | 0.460000 |
| 0.41 | 0.457975 |
| 0.42 | 0.455900 |
| 0.43 | 0.453775 |
| 0.44 | 0.451600 |
| 0.45 | 0.449375 |
| 0.46 | 0.447100 |
| 0.47 | 0.444775 |
| 0.48 | 0.442400 |
| 0.49 | 0.439975 |
| 0.50 | 0.437500 |

same accuracy that occurs in Tables 1,2 and 3. From Tables 4 and 5, it is observed that if the variables, $R$ and $a$, are held constant, $M_{B}$ decreases when $c$ is increased. If the variables, $R$ and $c$, are held constant, $M_{n}$ increases when $a$ is increased.

Parme and Paris ${ }^{2}$ have derived a formula for the fixed-end moments due to prestressing for an interior span. The cable profile is made of parabolas and is symmetrical about the centerline of the interior span. The equation which they derived, expressed in the notation of Fig. 16 is

$$
\begin{equation*}
M_{A}=M_{B}=+\frac{F y(1-a)}{1.5} \tag{49}
\end{equation*}
$$

Beam sign conventions apply to Eq. 49.
(A)


Fig. 16-Symmetrical Parabolic Cable Profile for Interior Span

Next an elastic design example will be given illustrating the use of Tables 4 and 5. Fig. 17 shows a three span continuous beam ( 50.0 ft . $-60.0 \mathrm{ft} .-50.0 \mathrm{ft}$.) subjected to a uniform load of 600 lbs . per foot and a uniform live load of 1000 lbs . per foot. For simplicity, no partial span loading of the live load is used. Fig. 18 shows the maximum and minimum moment curves for this beam.

Fig. 19 shows the selected beam which has a kern distance of 7.0 inches. The limiting zone for the center of pressure on concrete is

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## Table 5



|  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
|  | 45 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
|  |  |  |  | 4593 | 4.462 |  |  |  |  |  | 5170 |  |  |  | 463 |  |
| . 03 | .03 4307 | 4374 | 4442 | 4510 | 4578 | 4.647 | 4717 | 48 | ${ }_{4857}^{49}$ | 5013 | 5085 | 5137 |  | 5 |  |  |
| . 04 | -04 4225 | 5292 | 4359 | 4427 | 4495 | 456 | 4633 | 4703 | 4773 | 4844 | 4915 | 498 | 5059 | 5132 | 5205 |  |
|  | 406 |  | 4277 |  | 4423 |  |  |  |  |  |  | 003 |  |  |  |  |
|  | 3982 | 4048 | 4115 | 4182 |  | 43 | 4386 |  |  |  |  |  |  |  |  |  |
| . 08 | - 83 | 3968 | 4034 | 4101 | 4168 | 4236 | 4304 | 4373 | 4542 | ${ }_{4512}^{434}$ | 4582 | 453 | 4724 | 4796 | 486 |  |
| . 09 | 9 3823 |  | 5 4 | 4021 | 4088 | 4155 | 4223 | 4292 |  | 44 | 450 | 4571 |  |  | 4785 |  |
| . 110 | 32 |  |  |  |  |  | 40 |  |  |  |  |  |  |  |  |  |
| . 12 | 23587 |  | 3717 | 3783 | , | 3916 | 4053 | 4051 | 4119 | ${ }_{4188}^{428}$ | 4257 |  |  |  |  |  |
| - | 3510 | 3574 | 36 | 3705 | 3771 | 3837 | 3904 | 3972 | 4040 | 4108 | 4179 | 4247 | 4316 | 4387 | 1.45 | 45 |
| $\cdot 1$ | 343 | 3497 | 3562 | 3627 | 3693 | 3759 | 3826 | 3893 | 396 | 4029 | 4098 | 4167 | 4236 | 4307 | 4378 |  |
| . 115 |  |  |  |  |  |  | 374 |  | 3882 |  |  |  |  |  |  |  |
|  | 7320 | 5268 |  |  |  |  |  |  | 804 |  |  |  |  |  | 4,218 | 42 |
| . 18 | 83130 | 3193 | 3257 | 3321 | 3386 | 3527 | 3593 | 3368 | 3727 <br> 365 | 3794 | 3785 | 21 | 4000 |  | 4139 | 413 |
| . 19 | 93055 | 3118 | 3182 | 3246 | 3310 | 3375 | 3441 |  |  | 3640 |  |  | 364 | 3913 |  |  |
|  | 13981 | 3044 | 31.07 | 3171 |  | 3300 | 3365 |  | 4497 | 556 | 3631 |  | 767 | 3836 | 3905 |  |
| . 21 |  |  |  |  |  |  |  |  | 3122 |  |  |  |  |  |  |  |
| -23 | 2762 | 2824 | 2887 | 2950 | 3013 | 3151 | 13216 | ${ }_{3207}^{3281}$ | 33272 | 3433 | 3480 | 3547 3472 | 3615 | 3683 | 3752 | 3821 |
| . 24 | 2690 | 2752 | 2814 |  | 29 | 3004 | 30 | 3133 | 3198 |  |  |  | 3464 | 3532 | 3600 | 3749 |
| - 26 | 254, |  |  |  |  |  | 29 |  | 31 |  |  |  |  |  |  |  |
|  | 2477 | 2538 | 2600 | 2662 | 2724 | 2787 | 2851 | 2915 |  |  |  |  |  | 矿 | 351 |  |
|  | 2407 | 24,68 | 2529 | 2592 | 2653 | 2716 | 2779 | 3843 | 2907 | 2972 | 3037 | 3103 | 3169 |  |  |  |
| . 29 | 2338 | 98 | 2459 | 2521 | 2583 | 2645 | 2708 | 27 | 2836 | 2900 | 2965 |  |  |  | 3230 |  |
| . 30 | 2269 | 2329 | 2390 | 2451 | 2513 | 2575 | 2638 | 2701 | 2765 | 2829 | 2894 | 2959 | 3025 | 3091 | 158 |  |
| . 31 | 2200 | 2260 | 2321 | 2382 | 24.43 | 2505 | 2568 |  | 2694 |  | 2823 |  |  | 3019 | 086 |  |
| $\overbrace{-3} \cdot 3$ | 2132 | 192 | 2252 | 2313 | 2374 | 24 | 2498 | 2561 | 2624 |  |  | 2817 | 2882 |  |  |  |
|  | 2065 | 2124 | 2184 | 2245 |  | 2367 | 2429 | 2492 | 2555 | 2618 | 2682 | 2747 | 2812 | 2877 | 2943 | 3010 |
| -34 | 2000 | 2057 | 2117 | 2177 | 2239 | 2299 | 2361 | ${ }_{2}^{2423}$ | 24.86 | 2549 | 2613 | 2677 | 2742 | 2807 | 2873 |  |
|  | 1865 |  |  | 2043 | 2103 | 2164 |  |  |  |  |  |  | 2603 | 2857 |  |  |
| . 37 | 71800 | 1858 | 1917 | 1977 | 2037 | 2097 | 2158 | 2210 | 2282 | ${ }_{234}^{24}$ | 2.07 | 2471 |  | 2599 |  |  |
| -38 | \% 1735 | 1793 | 1852 | 1911 | 1971 | 2031 | 2092 | 2153 | 2215 | 2277 | 2340 | 24.03 | 2467 | 2531 |  |  |
| - 40 | 1606 | 16 | 17 | 1846 |  |  | 2026 | 208 | 21 | 2210 |  |  |  |  |  |  |
| . 42 | 1543 | 1600 |  |  |  | 1835 | 18 |  | 2017 |  | 2140 | 2203 |  |  |  |  |
|  |  | 1537 | 1595 | 1653 | 1712 | 1771 | 1831 | 1891 | 19 | 201 |  |  |  | 226 |  |  |
| .43 | 1417 | 1474 | 532 | 1590 | 1648 | 1707 | 1767 | 182 | 88 |  | 2010 | 2072 | 2134 |  | 2261 |  |
| . 4.45 | 1355 |  |  |  |  |  |  |  |  | 1884 |  | 2007 |  | 2132 | 5 |  |
| . 46 | 1233 | 1289 | 1346 | 1403 |  | 1519 |  |  |  |  |  |  |  |  | 6 |  |
| . 48 | 1172 | 1228 | 1285 | 1342 | 1399 | 1457 | 1516 | 1575 | 1634 | 1694 | 1755 | 1816 |  |  |  |  |
| -48 | ${ }_{1053}^{1112}$ | ${ }_{1108}^{1168}$ | ${ }_{1124}^{124}$ | 1281 |  |  | 1454 | 513 | 572 | 1632 | 69 |  | 1814 |  |  |  |
|  | 0994 | 1049 |  |  |  |  |  | 1452 | 1511 | 1570 | 1560 |  |  |  |  |  |
|  | 0935 | 0990 | 1046 | 1102 | 1158 | 21 | 273 | 1331 |  |  |  |  |  |  |  |  |
| . 53 | 0877 | 0932 | 0987 | 1043 | 1099 |  | 1213 |  | 1329 |  |  |  |  |  |  |  |
| .53 <br> .54 <br> .55 <br> 5 | 0820 | 0874 | 0929 | 0985 |  |  |  | 1212 |  |  |  |  |  |  |  |  |
| -54 | 0763 | $0817$ | OB72 | $0927 \mid$ | $10983$ | $1039$ | 1096 |  |  |  |  |  |  |  |  |  |
|  | 0550 | 0704 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
|  | 0595 | 0648 | 02 | 0757 |  |  |  |  |  |  |  |  |  |  |  |  |
| $\begin{array}{r}.58 \\ \hline .59 \\ \hline 80\end{array}$ | 0540 | 0593 | 0647 |  |  |  |  |  |  |  |  |  |  |  |  |  |
| $: 59$ | ${ }_{0}^{1,31}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |

Note: Above values must be multiplied by $10-4$


Fig. 17-Design Loads for Three Span Continuous Beam
also designated. No tension stress is permitted in this beam.

A concordant cable consisting entirely of parabolas will be determined for this beam. For this design a prestressing force of 450 kips will be used. By inspecting Fig. 19, the following values have been chosen for span $A B: a=0.42, c=0.08$, $R=0.4, y=1.073 \mathrm{ft}$.

From Table 4, when $a=0.42$, the value is 0.455900 and from Table 5 when $a=0.42$ and $c=0.08$, the value is 0.4373 . Hence the constant to calculate the fixed-end moment at end $B$ of span $A B$ is $0.4373+0.4$ ( 0.455900 ) 0.6197 . Thus the fixedend moment is

$$
\begin{aligned}
M & =0.6197 F y \\
& =0.6197(450)(1.073) \\
& =300 \mathrm{kip}-\mathrm{ft} .
\end{aligned}
$$

The stiffness of span $A B$ at joint $B$ is a $3 E I / L$ member and due to symmetry of loading, the stiffness of span $B C$ is a $2 E I / L$ member. Hence, the distribution factors in the moment distribution analysis is 0.643 for member $B A$ and 0.357 for member $B C$.

Fig. 19 shows that the cable is located 0.644 ft . above the centroid of the beam. For the cable to be concordant, the final moment must equal $450(0.644)=290 \mathrm{kip}-\mathrm{ft}$. The final moment is now known and the fixed-end moment in span $A B$ is known. Now a cable profile and a fixed-end moment in span $B C$ must be chosen such that the cable will be concordant. Let $v$ be the unknown fixed-end moment in span $B C$ and solve for $v$ using moment distribution (see Fig. 20).
Thus

$$
\begin{aligned}
+290 & =+v+(300-v) 0.357 \\
v & =284 \text { kip. } \mathrm{ft} .
\end{aligned}
$$

From Eq. 23, the cable profile in span $B C$ is determined.

$$
\begin{aligned}
v= & 284=\frac{F y(1-a)}{1.5}= \\
& \frac{450 y(1-a)}{1.5}=300 y(1-a)
\end{aligned}
$$

The value of $a$ is selected to be 0.08 . Hence,

$$
y=\frac{284}{300(0.92)}=1.03 \mathrm{ft}
$$

Fig. 19 shows a concordant cable for the beam and presents an elastic design solution.

## CONCLUSION

Data presented in Tables 1 and 4 are parabolic data. Also data presented in Tables 3 and 5 are parabolic in both the $a$ and $c$ directions. Data presented in Table 2 are parabolic in the $c$ direction and linear in the $a$ direction. When data are


Fig. 18-Maximum and Minimum Moment Curves for Three Span Continuous Beam


Fig. 19-Limiting Zone and Cable Profile for Three Span Continuous Beam


Fig. 20-Moment Distribution Calculation to Determine Concordant Cable
parabolic, exact values may be calculated by use of parabolic interpolation coefficients which may be developed by using Lagrange's Interpolation Polynominals. For those not familiar with this method and for those who do not feel that this refinement is necessary, straight line interpolation will give accuracy equal to or less than 1 in the fourth decimal place.
Equations and data for fixed-end moments for the end-spans of continuous units have been computed and derived for a beam fixed at one end and simply supported at the other end. These boundary conditions must be considered when using these fixed-end moments in the structural anlaysis of the beam.

The tables and equations presented in this paper offer the designer great flexibility in selecting his cable profile and a rapid method of calculating fixed-end moments due to prestressing for further use
in structural analysis.

## ACKNOWLEDGEMENT

The writers would like to acknowledge the Computation Center at the University of Florida, Gainesville, Florida for making possible the computation of the tables presented in this paper. Thanks is due to Dr. A. A. Toprac for reading this material and making constructive suggestions when it was submitted as a Master's Thesis.

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Portions of this paper were presented to The University of Texas in partial fulfillment of the requirements for the degree of Master of Science in Civil Engineering.


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