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NBSLD, the Computer Program for Heating and Cooling Loads in Buildings

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NBSLD, the Computer Program for Heating and Cooling Loads in Buildings

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Preface

This document comprises the engineering manual for the computer program called the National Bureau of Standards Load Determination Program hereafter referred to as NBSLD. Presented herein are the algorithms for the exact calculation methodology that was developed in the Thermal Engineering Section of the National Bureau of Standards to determine accurate heating and cooling loads for the thermal design of buildings. NBSLD, which is based upon the methodologies presented in this publication, has been available for some time for the purpose of evaluating various building constructions and systems. The program was originally developed as a research tool because none of the commercially available programs had features or the sophistication to enable the evaluation of unconventional designs. NBSLD has been an indispensable tool for studies of numerous HUD housing systems, constructions of the Defense Department and the General Services Administration where non-conventional design conditions had to be evaluated.

As the existence and the capability of NBSLD became known, numerous requests were made to NBS to release the program for public use. This publication is in response to that request. Hopefully, engineers will be able to adopt some of the computational schemes described in this publication to their own programs. A complete Fortran program is attached, although NBS does not claim that the program is optimum from the standpoint of the computer memory allocation or computational economy. It will take additional improvements before the program becomes optimum from those viewpoints. The program documentation is being made available at

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this time so engineers can use it for accurate load determination as they seek to conserve energy through improved thermal design of buildings.

The author would appreciate receiving reader's comments with respect to the accuracy of this text.

It should be mentioned that some of the subroutine algorithms listed in this publication have already been published in the well known ASHRAE booklet entitled "Procedure for Determining Heating and Cooling Loads for Computerized Energy Calculations". These subroutines were compiled by the author who served as the chairman of the Subcommittee on Heating and Cooling Load Calculations of the ASHRAE Task Group on Energy Requirements for Heating and Cooling of Buildings. The ASHRAE publication, however, contains several critical errors, which have been corrected for use in this volume.

The author is greatly indebted to Dr. J. E. Hill and Mr. J. P. Barnett for their thorough editing of the text, to Mrs. Sharon D. Crampton for her skill and patience in typing the manuscript and to Mr. F. J. Powell for his encouragement to produce the document.

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National Bureau of Standards Heating and Cooling
Load Determination Program

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A comprehensive computer program called NBSLD, the National Bureau of Standards Load Determination program, has been developed at NBS to reflect the time change of the many building parameters which are pertinent to accurate estimation of energy usage for heating and cooling. Current status of heating and cooling load techniques is reviewed. Of general interest are unique features of NBSLD which are not available in existing computer programs. A summary of various subroutines of NBSLD is given along with the detailed procedures for them. These subroutines constitute the recommended subroutine algorithms of the ASHRAE Task Group on Energy Requirements. Complete Fortran listing of NBSLD and data preparation forms are given for those who wish to use the program. The NBSLD computation is on the basis of the detailed solution of simultaneous heat balance equations at all the interior surfaces of a room or space. Transient heat conduction through exterior walls and the interior structures is handled by using conduction transfer functions. The use of heat balance equations, although time consuming in calculation, can avoid the vagueness and uncertainties inherent in the more popularly used weighting

factor approach. In addition, it is more accurate for a specific build-
design.

Key words: ASHRAE Task Group on Energy Requirements; conduc-
tion transfer functions; heating and cooling load; National Bureau of
Standards Heating and Cooling Load Computer Program.

1. Introduction

Numerous studies in recent years on the matter of energy shortage lead one to believe that the U. S. demand for energy will very shortly outstrip her power generating capacity and fossil fuel supply. According to a recent report of the Stanford Research Institute^{1/}, space heating and cooling for residential and commercial buildings amounts to approximately 20% of the total energy consumed in the United States, which was 60 trillion Btu per year in 1968. Moreover, recent and frequent blackouts and brownouts in the east coast region of the United States are good indications that the electric power demand for summer air conditioning exceeds for certain times, the capability of supply and distribution by the power companies.

It is in this context that new and accurate methodology for energy calculations is most crucial for the design and analysis of the performance of space heating and cooling systems. This is especially true in view of the fact that the current load calculation procedures could lead to the over-design of heating and cooling equipment and imprudent use of energy.

It is generally accepted that buildings can be designed to be energy effective if their thermal insulation is increased; window size, air leakage, and lighting levels decreased; shading devices properly installed; heating and cooling systems adequately designed, installed, and maintained; and their heat storage capability most fully utilized. These energy saving features, however, must be considered with reference to numerous constraints, such as added costs for material, construction and maintenance,

conformance to local building codes, occupancy life style aesthetics, construction practices, and availability of equipment.

In spite of these constraints, there is sufficient engineering information and technical basis that exist today to warrant extensive studies on various design alternatives for heating and cooling the building to minimize the wasteful use of energy. Design and operation of heating and cooling systems based upon conventional steady-state calculations, for example, usually result in oversizing of equipment and overheating or cooling of the space to be controlled. An over-design system usually operates at lower efficiency and needs more material (consequently more energy) to produce it, thus creating a vicious cycle.

One effective way to design the heating and cooling systems which is optimum from the standpoint of energy consumption, peak power demand and many practical constraints mentioned above, is to study the building thermal performance by using accurate simulations. Because of the use of computer simulations make it possible to evaluate the sensitivity of various design alternatives on the net energy usage, they can be a very effective tool in the design process. In order for such design studies to be conducted on the computer however, the computer program to be used should be very comprehensive and should indicate the proper response to the change of the many parameters which are pertinent to energy usage. The intent of this document is to present a more detailed calculation methodology than is generally used to make it possible for engineers to reduce the area of approximation, where this is considered desirable, by a rigorous computer simulation of building systems, which consider and take into account most of the variables that affect the building and

system operation.

Refined and sophisticated calculation procedures unfortunately are both time consuming and expensive. Without the use of advanced computer methods, they are literally impossible. The development of such calculation procedures can only be justified on the basis that the more accurate calculation will result in overall savings in energy usage and owning and operating costs and consequently in total life cycle cost due to better design of the building systems, more precise sizing of the equipment, and more carefully controlled operation of the heating and cooling system. There are many indications that such a justification is well warranted.

2. Fundamentals of Heating and Cooling Load Calculation

Calculation of the energy requirements of the heating and cooling system of a building involves three major steps which may be carried out simply to achieve approximate results, or with increasing degrees of complexity and sophistication as more accurate and more refined determination of system performance is required. First, is the calculation of heat loss or heat gain to the space which is heated or cooled. Second, is the determination of the heating and cooling load imposed on the system. Third, is the calculation of the energy input to all of the system components to satisfy that load.

The ASHRAE Handbook of Fundamentals^{2/} contains the basic information whereby the heating and cooling load of a building may be calculated. Customarily such load calculations are made for the so-called "design conditions" for sizing the equipment and developing the design of the heating and cooling system. However, the "design conditions" normally exist only

for a very few hours, if at all, during a heating or cooling season. Consequently, the actual day-by-day and hour-by-hour heating and cooling load for energy consumption is quite different from that for the design condition. Thus the heating and cooling load calculation for the purpose of estimating "energy requirements" must reflect the actual weather conditions rather than a design condition.

Various methods have been developed in the past such as the "degree day" method or "bin" method for proportioning the design load, to obtain approximate monthly, daily, or hourly loads and consequently provide a basis for determining energy requirements. Insofar as such methods are based on valid approximation procedures and checked against actual operating experience, they provide the base for simplified determination of energy requirements acceptable for the needs of most engineers.

In this section a review of the rigorous methods of calculating heating and cooling loads by means of solving heat balance equations at all the interior surfaces of a room or space is given. Also described are approximate methods in which weighting factors are developed after the heat balance equations have been solved for one set of conditions. The NBSLD calculations follow the rigorous method while the ASHRAE Task Group procedures use the weighting factor method.

In NBSLD the transient heat conduction through exterior walls of the room or space is handled by using conduction transfer functions. The use of heat balance equations at the interior surfaces, although more time consuming, can avoid the vagueness and uncertainties inherent in the weighting factor approach. In addition, it is more accurate for a specific building design.

2.1 A Rigorous Method of Calculating Heating and Cooling Loads

A cooling (heating) load is of course the amount of energy that is transferred to (from) the room and simultaneously removed (added) by the conditioning equipment at any given time of interest. To calculate this quantity directly requires a rather laborious solution of energy balance equations involving the room air, surrounding walls, infiltrating and ventilation air, and internal energy sources. The principle of calculation can be demonstrated by considering a fictitious space that is enclosed by 4 walls, a ceiling and floor, and having infiltration air as well as normal internal energy sources. The six equations that govern energy exchange at each inside surface at a given time t are:

$$q_{i,t} = h_{ci} (t_{a,t} - t_{i,t}) + \sum_{\substack{j=1 \\ j \neq i}}^m g_{ij} (t_{j,t} - t_{i,t}) + RS_{i,t} + RL_{i,t} + RE_{i,t}$$

for $i = 1, 2, 3, 4, 5, 6$

where

m = number of surfaces in the space

$q_{i,t}$ = rate of heat conducted into surface i at the inside surface at time t

h_{ci} = convective heat transfer coefficient at interior surface i

g_{ij} = radiation heat transfer factor between interior surface i and interior surface j

$t_{a,t}$ = inside air temperature at time t

$t_{i,t}$ = average temperature of interior surface i at time t
 $t_{j,t}$ = average temperature of interior surface j at time t
 $RS_{i,t}$ = rate of solar energy coming through the windows and absorbed by surface i at time t
 $RL_{i,t}$ = rate of heat radiated from the lights and absorbed by surface i at time t
 $RE_{i,t}$ = rate of heat radiated from equipment and occupants and absorbed by surface i at time t

The equations governing conduction within the six slabs cannot be solved independent of the above equations since the energy exchanges occurring within the room affect the inside surface conditions which in turn affect the internal conduction. Consequently, one is faced with solving six equations simultaneously with the governing equations of conduction within six slabs in order to calculate the cooling load at time of interest

($Q_{L,t}$) which would be given by:

$$\begin{aligned}
 Q_{L,t} = & \sum_{i=1}^6 h_{ci} (t_{i,t} - t_{a,t}) + \rho C G_{L,t} (t_{o,t} - t_{a,t}) \\
 & + \rho C G_{v,t} (t_{v,t} - t_{a,t}) + RS_{a,t} + RL_{a,t} + RE_{a,t}
 \end{aligned}$$

where

ρ = air density
 C = air specific heat
 $G_{L,t}$ = mass flow rate of outdoor air infiltrating into the space at time t
 $t_{o,t}$ = outdoor air temperature at time t

$G_{v,t}$ = mass rate of flow of ventilation air at time t
 $t_{v,t}$ = ventilation air temperature at time t
 $RS_{a,t}$ = rate of solar heat coming through the windows and convected into the room air at time t
 $RL_{a,t}$ = rate of heat from the lights convected into the room air at time t
 $RE_{a,t}$ = rate of heat from equipment and occupants and convected into the room air at time t

A rigorous approach such as this for calculating cooling load would be practically impossible if it were not for the speed at which such computations can be done by modern digital computers. Even so, there are very few computer programs in use today where instantaneous cooling loads are calculated in this exact manner. The concept, however, has been presented previously by Stephenson and Mitalas^{3/}, and by Buchberg^{4/}.

Not to be ignored is the effect of air temperature deviation from some prescribed set point. This set point is the temperature for which the cooling (heating) load calculation is made and for which the design capacity of the cooling (heating) apparatus is usually selected. A recent study by Mitalas and Stephenson^{5/} shows that actual heat extracted from the space is considerably smaller than the cooling load calculated on the basis of a constant space temperature. This is due to the thermal storage effect of the building structure and internal furnishings. Figure 1 shows a result from that study and as can be seen, the calculated cooling load peaks at values considerably higher than the measured heat extraction rate.

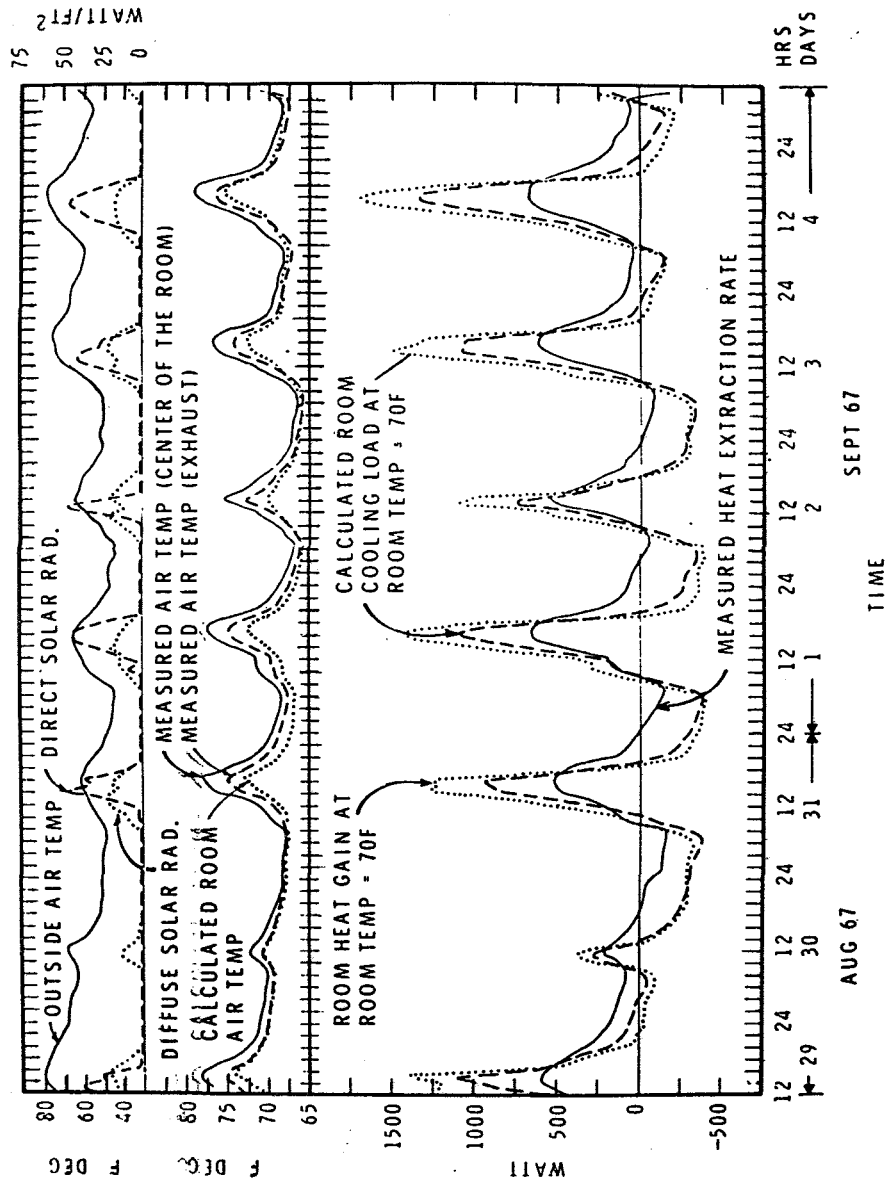


Figure 1 Measured and Calculated Thermal Performance in an Ottawa Office Building (Reference 4)

2.2 Approximate methods of Calculating Heating and Cooling Loads

Since the exact solution technique is extremely time consuming especially for the calculations done for a period of 8760 hours (one year), the ASHRAE Task Group on Energy Requirements recommends a transfer function concept to simplify the calculation procedure. The transfer function concept was first introduced by Mitalas and Stephenson^{3/} using what they called room thermal response factors. Their procedure is as follows: the room surface temperatures and cooling or heating load are first calculated by a rigorous method as described in the previous section for several typical constructions representing offices, schools and dwellings of heavy, medium and lightweight construction. In these calculations, the components such as solar heat gain, conduction heat gain, or the heat gain from the lighting, equipment, and occupants are simulated by pulses of unit strength. The transfer functions are then calculated as numerical constants which represent the cooling load (or heating) corresponding to the input excitation pulses. Once these transfer functions are determined for a number of typical constructions, they are assumed to be independent of input pulses and the determination of cooling loads (or heating) is possible without resorting to the rigorous calculations. The calculation required is, instead, simple multiplication of the transfer functions by a time-series representation of heat gain and the subsequent summation of these products, which can be carried out on a small computer with little effort.

Another way to shorten the computational effort for energy calculations is to determine regression parameters by fitting a simple algebraic equation to the results of rigorous calculations, which had been obtained not for an entire year but for a limited period in the year, such as for the months of January and/or July. Once this regression equation is determined with sufficient accuracy, an energy estimate is made by superimposing the weather conditions of the other months onto the relationships just determined.

An example of this approach is illustrated in Figure 2, which depicts the daily total heating and cooling load plotted against the daily average outdoor air temperature. This plot is a result of a lengthy rigorous calculation performed on a typical apartment in Jersey City using the annual hourly weather conditions that occurred in 1949. The straight lines superimposed on the figures were the least square regression lines that best fit the calculated loads for January and July. It was clear from this figure that the exact calculations for other months would not be necessary, at least for the purpose of determining daily loads, since the regression relationship determined from the January and July calculations were sufficiently accurate that they could be extrapolated to the remainder of the year.

Depending upon the type of building and its heating and cooling system, a good correlation such as illustrated in Figure 2 may not be possible. Figure 3 shows, for example, a similar plot for a test office building whose heating and cooling load were measured in a research project conducted by Ohio State University. The scatter appears considerably larger than the calculated relationship obtained in the Jersey City study.

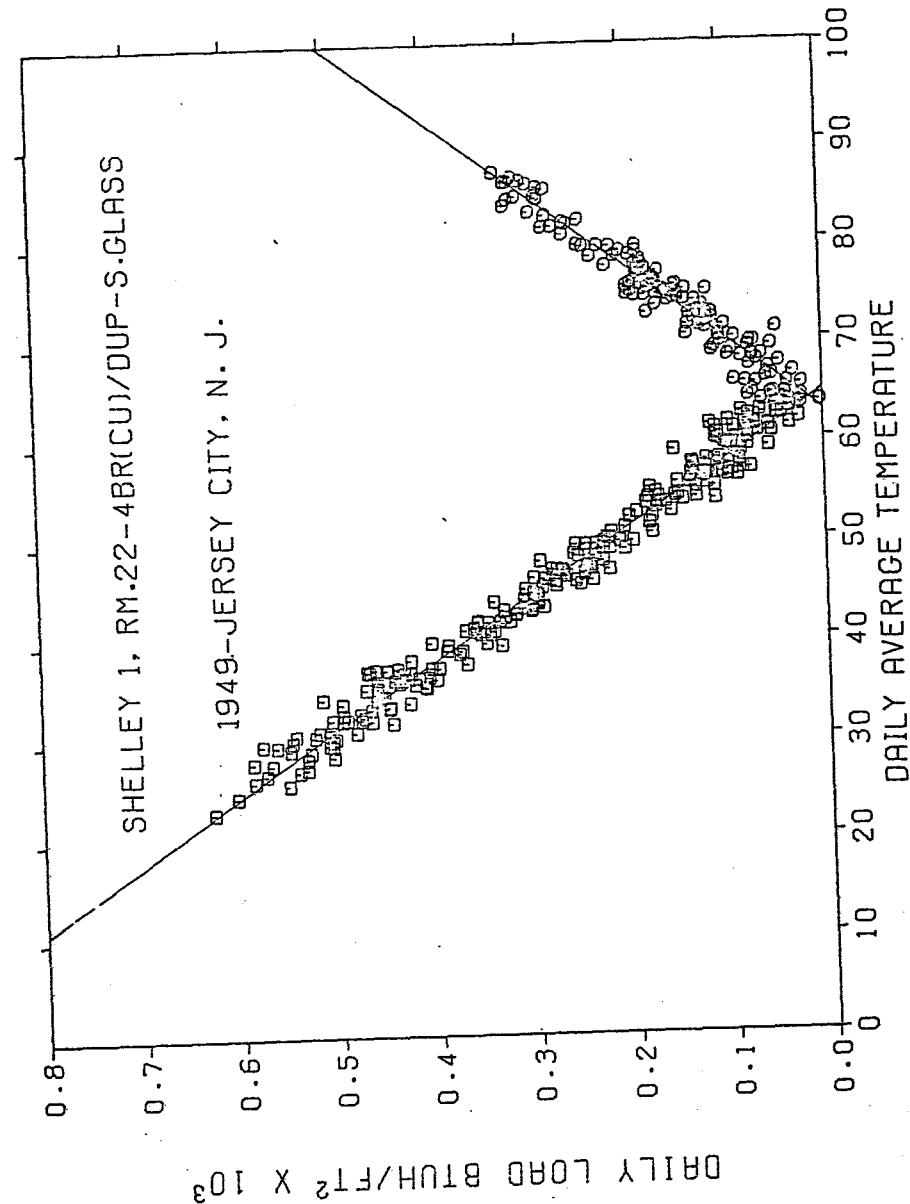


Figure 2 Calculated Daily Total Thermal Loads of a Jersey City Apartment Plotted Against Daily Average Temperatures

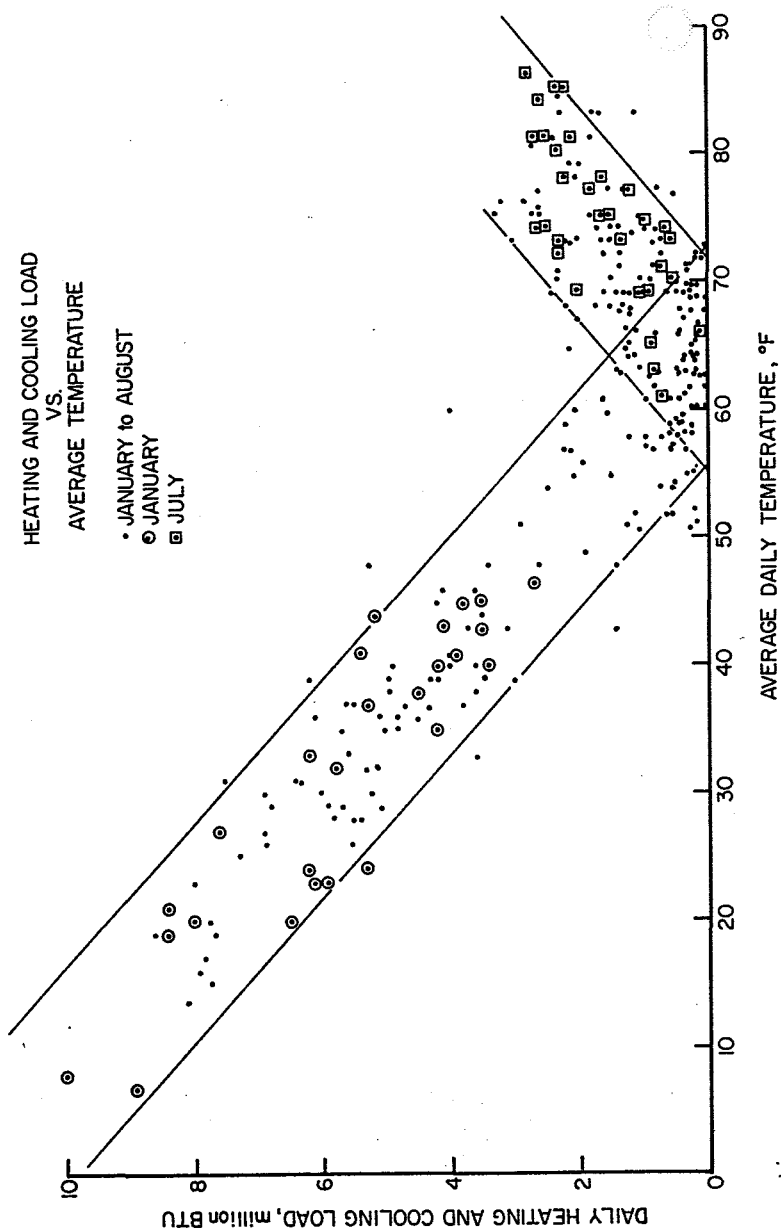


Figure 3 Measured Daily Total Thermal Loads of a Columbus Law Building Plotted Against Daily Average Temperatures

It is obvious that the thermal performance of commercial buildings are affected in a large way by the manner in which internal heat is generated. In addition to the normal dependence on outdoor temperature. The inclusion of one or more additional statistical parameters dealing with these internal heat gains should improve the correlation.

Recently one additional method of predicting the heating and/or cooling load or the indoor temperature as a result of the excitation parameters such as outdoor temperature, solar radiation and internal heat generation has been demonstrated by Kusuda and Tsuchiya^{6/} and further expanded by Kimura and Ishino^{7/}. The method uses the concept of equivalent thermal mass of a building and attempts to fit the observed input and output data into a linear differential equation. The initial results are promising. Figure 4 shows a comparison of predicted and measured room temperature and heat extraction rate for a simple one room test building studied by Kimura and Ishino.

Calculated room temperature in Figure 4 is obtained by the transfer functions derived from the measured data of August 19, while the heat extraction was calculated by the transfer functions derived from the measured values of August 13. The good agreement indicated in Figure 4 implies that the detailed calculation is needed only for a limited number of days to derive accurate transfer functions based upon the equivalent thermal mass of the particular building under consideration.

* Private communication with Professor C. F. Sepsey and J. Jones of the Ohio State University.

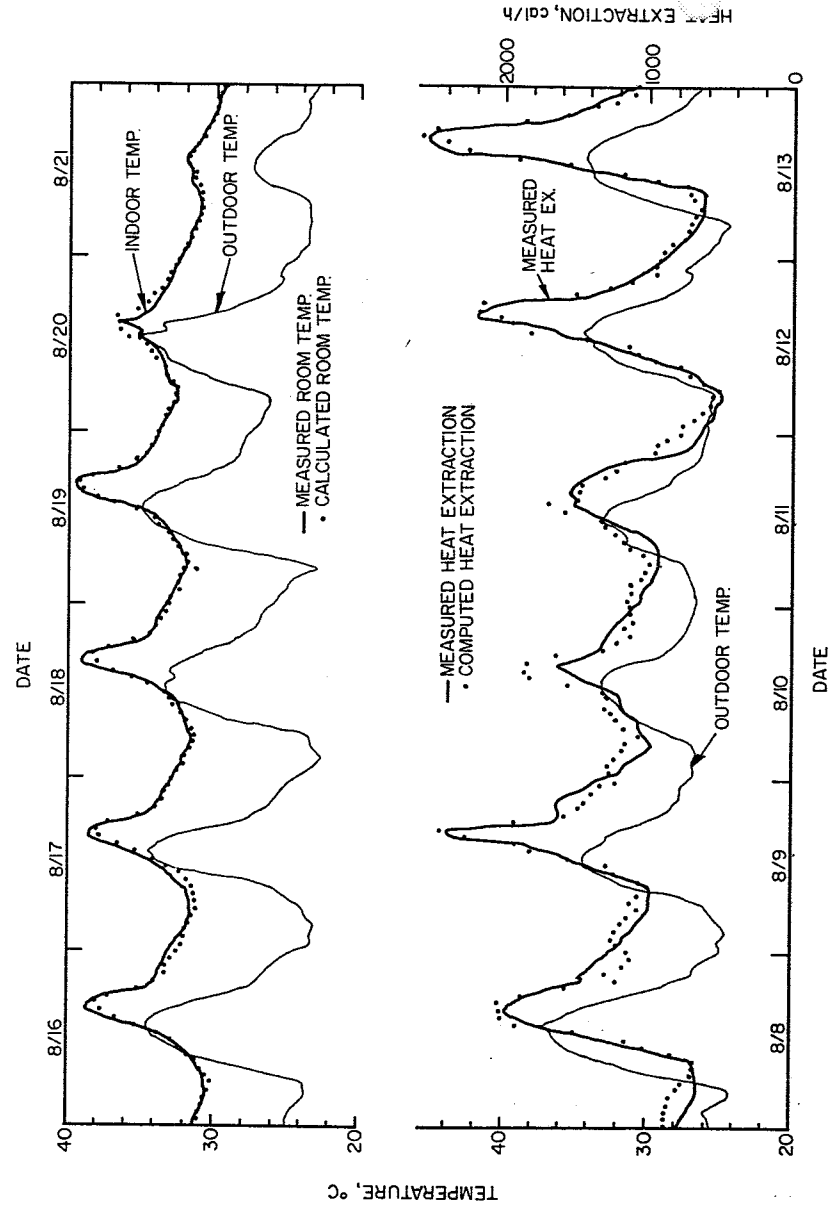


Figure 4 A Comparison of Measured and Calculated Thermal Performance of an Experimental Building (Courtesy of Professor K. Kimura)

3. Unique Features of NBS Load Calculation Computer Program

A comprehensive yet easy-to-use computer program for determining heating and cooling loads has been developed in the Thermal Engineering Section of the Center for Building Technology at the National Bureau of Standards. This computer program is based upon extensive information accumulated over the past decades in various phases of building heat transfer problems, and is intended to be used for the design of equipment and air conditioning systems as well as for estimates of building energy requirements.

The major reason why NBS developed this comprehensive program is that despite the existence of numerous load calculation programs currently available, most of them are not suitable for the analysis of building designs where non-conventional or innovative ideas on structures, heating and cooling systems and controls are employed. Some of the unique aspects of building and system design and operation that can be handled by or studied by using NBSLD are:

1. Inside-out construction of exterior walls where the thermal insulation is placed on the outside of the building shell as opposed to conventional walls having insulation on the inside. (These two walls could have the same U-value and yet their thermal response would be quite different.)
2. Effect of interior partition walls or floor-ceiling sandwich structures on the heat storage characteristics of the room,

3. Off-peak heating or cooling of buildings to shave the peak heating or cooling demand,
4. Evaluation of intentionally undersized heating and cooling equipment by calculating the room temperature and humidity deviations from a design set point. (The results would indicate whether or not the indoor conditions would remain within acceptable limits.)
5. Evaluation of indoor thermal environment of various zones during the intermediate season, such as spring and autumn, when the heating or cooling requirements for these zones may not be in phase with that of the building as a whole. (This would apply to a case where a two-pipe system would be installed for example. The central system for the entire building might be switched to heating in late autumn and yet some zones, particularly those facing south may still require cooling. NBSLD can be used to determine the indoor thermal conditions of unheated or uncooled rooms.)
6. Use of solar energy for heating and cooling buildings as it relates to the thermal storage characteristics of the building,
7. Use of attic ventilation to reduce the cooling load since NBSLD can accurately predict attic temperatures,
8. Accurate determination of the need for heating and air conditioning in basement rooms,

9. Design of heating and cooling systems and equipment on the basis of intermittent operation, such as the shutdown of air conditioning facilities during the nighttime or weekends,
10. Effective use of natural air conditioning such as ventilation, shading, increased ceiling insulation and the subsequent determination of the requirements for mechanical cooling,
11. Effective use of planned ventilation to minimize a building heating load during the winter season,
12. Accurate evaluation of indoor comfort conditions based upon air temperature, humidity and mean-radiant temperature,
13. Determination of the condition whereby moisture condensation takes place along interior surfaces of a building.

Figure 5 depicts an overall calculation sequence to attain the hour by hour heating and cooling load of buildings. Shown in the double lined boxes are input data to be supplied whereas those in single lined boxes indicate calculations to be performed. The cycle indicators show the iteration cycles for the number of buildings, the number of rooms in a given building and the number of days for which the calculations are performed. More specific identification of the types of input data needed are listed in Figures 6, 7 and 8. The exact way the input data is put into the program is specified in Appendix C.

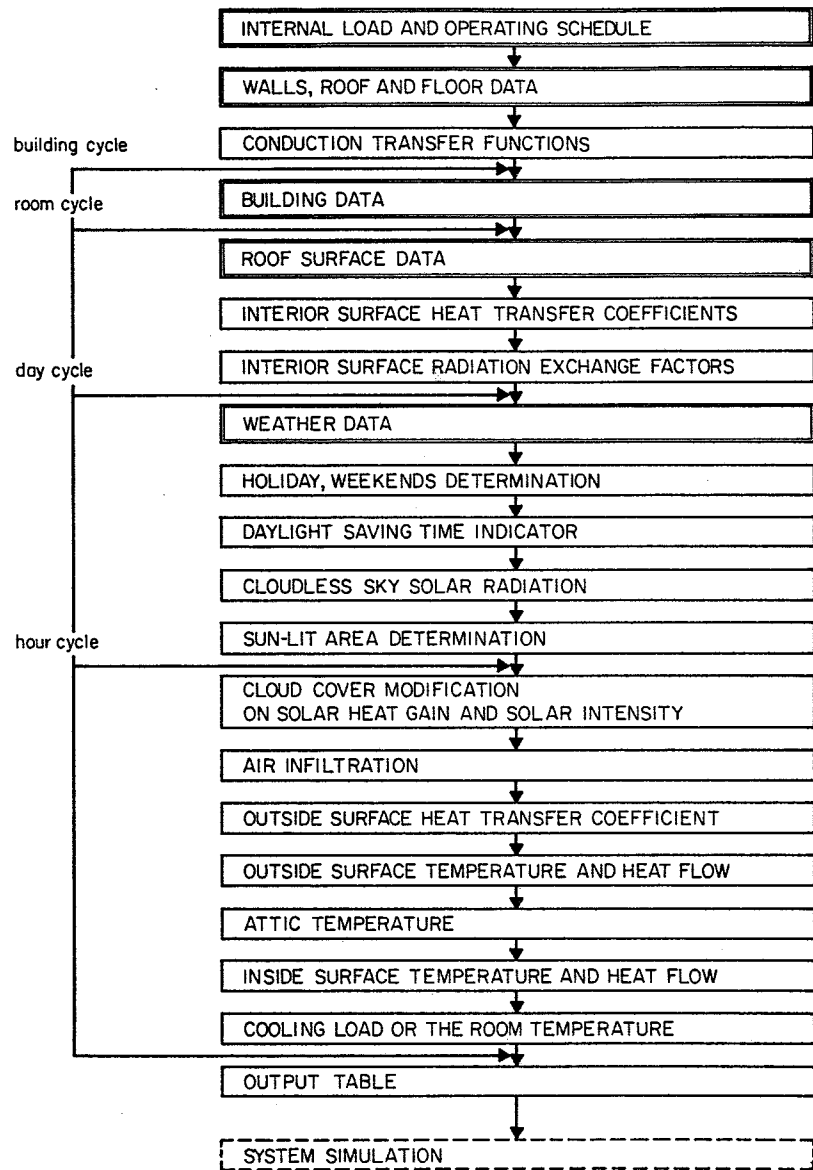


Figure 5 Calculation Sequence of NBSLD

INPUT - OPERATING DATA

- ELECTRIC POWER TO LIGHTS, WATTS PER SQUARE FOOT OF FLOOR (QLITY)
- HOUR BY HOUR LIGHTING SCHEDULE (QLITX)
- ELECTRIC POWER TO EQUIPMENT, WATTS PER SQUARE FOOT OF FLOOR (QEQFX)
- HOUR BY HOUR EQUIPMENT SCHEDULE (QEQUX)
- SUPPLY AIR RATE (CFMS)
- AIR LEAKAGE RATE (CFML)
- SUPPLY AIR TEMPERATURE (TS)

Figure 6

INPUT - BUILDING DATA

- ROOM NUMBER (ROOMNO)
- CEILING HEIGHT (H)
- ROOM LENGTH (L)
- ROOM WIDTH (W)
- NUMBER OF OCCUPANTS (QCU)
- OCCUPANT SCHEDULE (QOCUP)
- WINTER WINDOW OVERALL HEAT TRANSFER COEFFICIENT (UGLAS)
- GROUND FLOOR HEAT TRANSFER COEFFICIENT (UG)
- SUMMER INFILTRATION, AIR CHANGES PER HOUR (ARCHGS)
- WINTER INFILTRATION, AIR CHANGES PER HOUR (ARCHGW)
- TYPE OF HEAT TRANSFER EXPOSURES (ITYPE)
-ROOFS-WALLS-WINDOWS-DOORS-FLOORS
- TYPE OF RESPONSE FACTORS TO BE USED (IRF)
-HEAVY/LIGHT: ROOF, EXTERIOR WALLS, CEILING/FLOOR
PARTITION
- U VALUE OF THE EXPOSURE (U) (ONLY WHEN RESPONSE FACTOR
IS NOT CALCULATED)
- AREA OF THE EXPOSURE (A)
- ORIENTATION OF THE EXPOSURE (AZW) -N,E,S,W (ONLY FOR
EXTERNALLY EXPOSED SURFACES)
- WINDOW SHADING COEFFICIENT (SHADE)
- SOLAR HEAT ABSORPTION COEFFICIENT FOR THE EXTERIOR SURFACE
(ABSP)
- TIME INCREMENT OF TEMPERATURE DATA USED
- PROPERTIES OF BUILDING MATERIALS - THICKNESS - THERMAL
CONDUCTIVITY - DENSITY - SPECIFIC HEAT
- NUMBER OF SURFACES IN EACH WALL (NS,NW,NN,NE)

Figure 7

INPUT - WEATHER DATA

- LATITUDE (LAT)
- LONGITUDE (LONG)
- TIME ZONE NUMBER (TZN)
- MONTH (MONTH)
- DAY (DAY)
- ELAPSED DAYS SINCE JANUARY 1 (ELAPS)

- MAXIMUM TEMPERATURE OF THE DESIGN DAY (DBMAX)
- DAILY TEMPERATURE RANGE OF THE DESIGN DAY (RANGE)
- DESIGN INDOOR TEMPERATURE CONDITION (DBIN)
- DESIGN OUTDOOR WET-BULB TEMPERATURE (WBMAX)
- DESIGN INDOOR WET-BULB TEMPERATURE (WBID)
- DESIGN WINTER OUTDOOR TEMPERATURE (DBMWT)
- DESIGN SUMMER GROUND TEMPERATURE (TG)
- DESIGN WINTER GROUND TEMPERATURE (TGW)

Figure 8

4. General Description of NBSLD Subroutines

In order to perform the chain of calculations depicted in Figure 5, a number of subroutines were developed at the National Bureau of Standards, the algorithms of which have already been published through the ASHRAE Task Group booklet entitled "Procedure for Determining Heating and Cooling Loads for the Computerized Energy Calculations". This booklet, however, contained several errors which have been corrected and is attached to this report as Appendix A. NBSLD incorporates most of the revised ASHRAE algorithms as they are written; however, some of them have been combined or split to fit the overall computational scheme in NBSLD. Listed below is a brief description of the NBSLD subroutines with their specific reference to the ASHRAE algorithms in parentheses.

1. ABCD, ABCDP, ABCD2, ABCDP2, DERVT, GPF, MULT, RESF, RESFX, RESPTK: These routines are parts of the conduction transfer functions calculation package and are needed for the accurate evaluation of thermal time lag, damping, heat storage in exterior facing surfaces as well as the internal furnishings. (XYZ)
2. AIRCON: This routine is used to determine instantaneous values of the physiological indices for the space being studied such as ASHRAE's New Effective Temperature, Predicted Mean Vote (Fanger), Heat Stress Index, KSU Index, Resultant Temperature, Operative Temperature, and Index of Thermal Stress (Givoni). - not included in the text

3. ATTIC: Attic space temperature and heat conduction through the ceiling into the room below are calculated by this routine for the vented or non-vented attics. (ATTIC)
4. CCM: This routine modifies the solar radiation computed for a cloudless sky by instantaneous cloud cover data. (CCF)
5. DPF: This routine calculates dew point temperature of atmospheric air when the partial vapor pressure is known. (PSY)
6. DST: This routine determines whether a given data is in a daylight saving time zone. The information is needed for the proper assignment of the energy usage schedule. (DST)
7. FCTR: This routine determines radiation exchange factors between any two surfaces which are part of a given room. For the room of six interior surfaces, for example, thirty radiation exchange factors are calculated. These factors are used in turn to determine the rate of heat exchange by radiation between all the interior surfaces. (FIJ)
8. F: This routine calculates radiation heat exchange factors (form factors) between two adjacent rectangular surfaces which are normal to each other, such as that between the floor and wall. (FIJ)

9. FO: This routine calculates the surface heat transfer coefficients for externally exposed surfaces from weather data. (FO)
10. GLASS: This routine calculates solar heat gain through glass when the shading coefficient, orientation, and type of glass are given. (SHG)
11. HOLIDAY: This routine identifies the national holidays in the United States so that the proper holiday schedule can be used for the energy calculation. (HOLIDAY)
12. OUTSID: This routine calculates the outside surface temperature and heat gain into the wall or roof by taking into account solar heating, back radiation to the sky, convective heat loss to the ambient air and transient heat conduction within the wall or roof. (HEATW)
13. PSY1: This is a psychrometric routine that determines the thermodynamic properties of moist air when the dry-bulb temperature, wet-bulb temperature and barometric pressures are given. (PSY)
14. PSY2: This routine is similar to PSY1 except that the dew point temperature is required instead of the wet-bulb temperature. (PSY)
15. PVSF: This routine determines the saturated vapor pressure of atmospheric air as a function of temperature. (PSY)

16. RMTMP: This is the single most important subroutine of NBSLD since it determines the room temperature by solving matrix equations expressing a balance of heat gains, heat storage at the room surfaces, and cooling capacity of an air conditioning unit. However, the room temperature can be prescribed, in which case the routine will calculate the heating/cooling requirements to satisfy that prescribed temperature. (RMTMP)
17. ROOM: This routine reads in all the data required for the room heat transfer calculation such as dimensions, surface orientations, shading coefficients, surface solar absorptivity, etc.
18. SHG: This is the routine that calculates solar heat gain through glass. (SHG)
19. SOLVP: This routine solves the simultaneous linear algebraic equations that appear in RMTMP. (RMTMP)
20. SUN: Basic sun data such as solar angles, cloud cover, direct and diffuse radiation needed for solar heat gain and solar heating of the building exterior surfaces are calculated in this routine. (SUN, SOLAD)
21. TAR: This routine calculates transmission and absorption characteristics of glass. (TAR)
22. WBF: This is another psychrometric routine that calculates the wet-bulb temperature when the enthalpy of moist air and the barometric pressure are specified. (PSY)

23. WKDAY: This subroutine determines the day of week when the date and year are given. The information is needed for the proper selection of energy usage schedules which are dependent upon whether the day is a weekday or not. (WKDAY)

24. WD, WDX, DECODE, ERROR, WEATHE: The weather data tape 1440 supplied by the National Climatic Center, Asheville, N. C. is prepared in a format which cannot be readily applicable in most of the Fortran programs. These routines are therefore necessary to read the 1440 tapes and decode them into meaningful weather parameters, which in turn can be used by UNIVAC 1108 Fortran of the National Bureau of Standards. If there are some data which are unreasonable, the data will be replaced by the arithmetic average of two adjacent data by the ERROR routine. (CLIMAT)

All the subroutines in the program may be used to form a separate main program for a specific job. Following are sample usages of some of the subroutines.

1. Psychrometric Calculation

CALL PSY1 (DB, WB, PB, DP, PV, W, H, V, RH)

where inputs are DB = dry-bulb temperature

WB = wet-bulb temperature

PB = barometric pressure

outputs are DP = dew point temperature

PV = vapor pressure

W = humidity ratio

H = enthalpy

V = volume

RH = relative humidity

There is also a routine called PSY2 (DB, DP, PB, WB, PV, W, H, V, RH) in which the inputs are DB, DP and PB instead of DB, WB and PB. In many cases DB and RH are the inputs and the vapor pressure and dew point temperature or the wet-bulb temperature at standard barometric pressure are the outputs. A possible algorithm that could be desired, for example, might be:

PVS = PVSF (DB)

PV = PVS*RH/100

DP = DPF (PV)

Call PSY2 (DB, DP, PB, WB, PV, W, H, V, RH)

2. Solar Radiation

Recently there has been increased interest in the application of solar energy for heating of hot water. SUN and GLASS routines should be valuable for evaluating various solar collectors at different locations in the United States at different times of the year. A sample use of these routines may be shown for a solar collector having the following characteristics (Figure 9)

location: latitude = 45°, longitude = 73°

azimuth angle = 0° south

tilt angle = 30° from horizontal surface

area = 500 ft²

date = July 21

time = 4:00 p.m.

glass cover = double sheet - clear glass

U_R = overall heat transfer coefficient between the collector surface and the ambient, Btu/hr ft² °F*

U_W = overall heat transfer coefficient between the collector surface and water, which is being circulated under the collector plate, Btu/hr ft² °F

TWI = water temperature entering the collector, °F

TWL = water temperature leaving the collector, °F

GPM = water circulation rate in gallons per minute

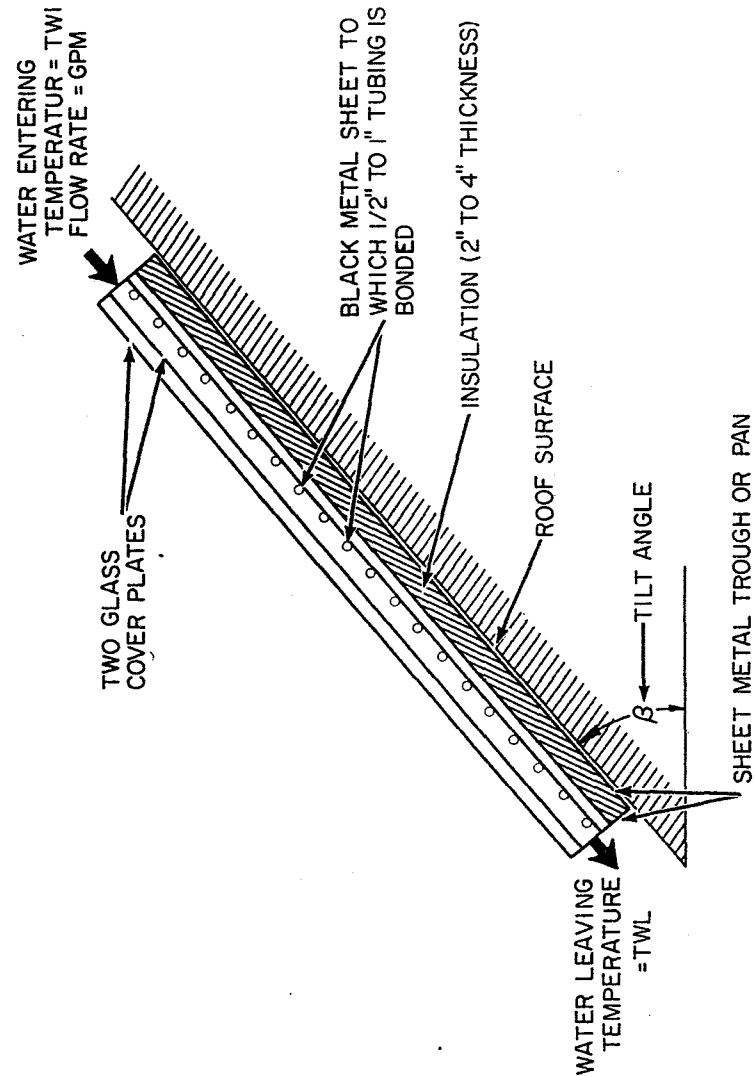


Figure 9 A Typical Flat Plate Solar Collector

* When the collector temperature is less than 150 °F, the typical value of U_R for a flat black collector with double glass cover may be 0.75. The value increases to as much as 1.5 when the collector temperature is in the neighborhood of 300 °F. A special computer program is available for estimating the value of U_R for various types of collectors.

It is assumed that the collector is well insulated around the edges and its bottom.

The algorithm for the use of SUN and GLASS routines would then be:

S(1) = LATITUDE = 30

S(2) = LONGITUDE = 73

S(3) = Time zone number = 5

S(4) = Elapsed days since January 1 = 202

S(5) = Time = 16

S(6) = IDST = daylight saving time index = 1

S(7) = Ground reflectivity = 0.2

S(8) = Clearness number = 1.0

S(33) = Cloud cover modifier = 1

S(9) = Azimuth angle of the collector = 0°

S(10) = Tilt angle of the collector = 30°

Call SUN

Call GLASS (SHDW, SHADE, GLASTP, GLAZE, SHG)

note: SHDW = sunlit area factor = 1

SHADE = shading coefficient = not applicable

GLASTP = 1/8" double strength glass (1)

GLAZE = double glazing (2)

SHG = solar heat gain - output

note: SUN and GLASS have S in common using the solar heat gain through the glass plate. When the edge heat loss is small, the following calculation may be used to estimate TS, collector surface and TWL, leaving water temperature from the collector:

$$TS = \frac{A*(SHG + UR*DB) + 500*GPM*TWI*Y}{UR*A + 500*GPM*Y}$$

where $Y = 1 - e^{-X}$

where $X = \frac{U_w*A}{500*GPM}$

5. NBSLD Logic Diagram

Figure 10 shows the way in which the various subroutines of NBSLD fit together in the usage of the program.

7. Appendix A

Subroutine Algorithms Prepared for the
ASHRAE Task Group on Energy Requirements

CLIMAT

A Procedure for Obtaining Climatic Weather Data

Climatic parameters needed for the hourly load calculations are:

- DB: Dry-bulb temperature, F
- DP or WB: Dew point or wet-bulb temperature, F
- CT: Cloud type
- TC: Total cloud amount
- V: Wind speed, knots
- DIR: Wind direction (clockwise from North), degrees
- PB: Barometric Pressure, in. Hg
- ID: Direct Solar Radiation, Btu per (hr) (sq ft)
- $I_{d,sky}$: Sky diffuse radiation, Btu per (hr) (sq ft)
- $I_{d,ground}$: Ground diffuse radiation, Btu per (hr) (sq ft)
- Rain, Snowfall: Precipitation data (Optional)

Hourly observations of these weather parameters for past years are available from the National Climatic Center either on magnetic tape or in card deck form. The hourly solar radiation data has been recorded for only approximately fifty stations throughout the United States (Table A-1). These data are, moreover, limited in their durations and completeness, and scarcely useful for the comprehensive energy analysis. On the other hand, the data series 144 includes the hourly observations of all of the parameters listed above except the solar radiation for more than 300 weather stations (Table A-2) covering a period of from ten to thirty years. Since the 144 series data are very much complete, it is

recommended that hour by hour energy calculations be made with this series of data supplemented by simulated solar radiation data. A method for simulating solar radiation will be described later in this booklet.

Because of the specific coding scheme employed by the National Climatic Center for storing the hourly weather data onto the magnetic tapes, the 144 series is not directly usable by the standard Fortran programs. Different computing systems such as IBM 370, CDC 6600 and UNIVAC 1108 have their own decoding routines to read these tapes. Included in this section is a listing of a Fortran program which illustrates a decoding scheme required to make use of the weather tapes. This listing was prepared by Mr. McKay, Data Reduction Section, of the National Climatic Center, Asheville, North Carolina.

For further information on the procurement of weather tapes and possible assistance in decoding, the following office may be contacted:

Mr. G. McKay or D. Calloway
National Climatic Center
Applied Climatology Division
Federal Building
Asheville, North Carolina 28801
Tel. (704) 254-0961 x203

Table A-1 Solar Radiation Data

Albuquerque, New Mexico
Apalachicola, Florida
Barrow, Alaska
Bethel, Alaska

Bismark, North Dakota
Blue Hill/Milton, Massachusetts
Boston, Massachusetts
Brownsville, Texas

Canton Island
Cape Hatteras, North Carolina
Caribou, Maine
Charleston, South Carolina

Cleveland, Ohio
Columbia, Missouri
Dodge City, Kansas
El Paso, Texas

Ely, Nevada
Fairbanks, Alaska
Fort Worth, Texas
Fort Worth, Texas

Fresno, California
Grand Lake/Granby, Colorado
Great Falls, Montana
Hatteras, North Carolina
Inyokern, California

Lake Charles, Louisiana
Lake Charles, Louisiana
Lincoln, Nebraska
Lincoln, Nebraska

Los Angeles, California
Madison, Wisconsin
Matanuska, Alaska
Medford, Oregon

Miami, Florida
Nashville, Tennessee
New York, New York
Oak Ridge, Tennessee

Omaha, Nebraska (North Omaha)
Phoenix, Arizona
Riverside, California
Santa Maria, California

Santa Maria, California
Sault Ste. Marie, Michigan
Seattle, Washington
Sterling, Virginia

Tucson, Arizona
Upton, New York
Wake Island
Washington, D. C.

SUN

An Algorithm to Find Solar Position, and Intensity of Direct Normal and Diffuse Radiation

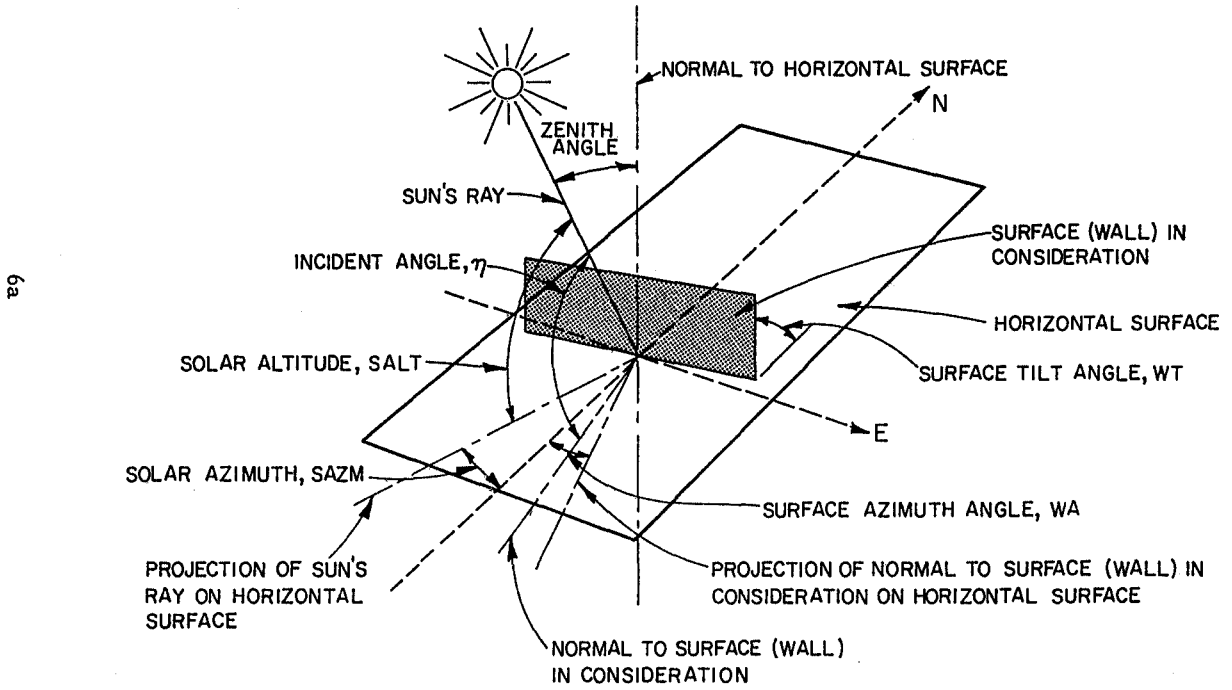


Figure A-1 Solar Angles for Tilted and Horizontal Surfaces

Figure A-1 DEFINITIONS OF SOLAR ANGLE

Data:

- L: Latitude, degrees, [+North]
[-South]
- λ : Longitude, degrees, [+West]
[-East]
- TZN: Time zone number (hours behind Greenwich mean time),
(see Figure A-3 and Table A-4)
- d: Date, days (from start of year), (1 - 366)
- t: Time, hours (after midnight), (0 - 24)
- DST: Daylight saving time indicator (Output of DST),
0 for standard time and 1 for daylight saving time
- ρ_g : Ground reflectivity
- CN: Clearness number (see Figure A-4)

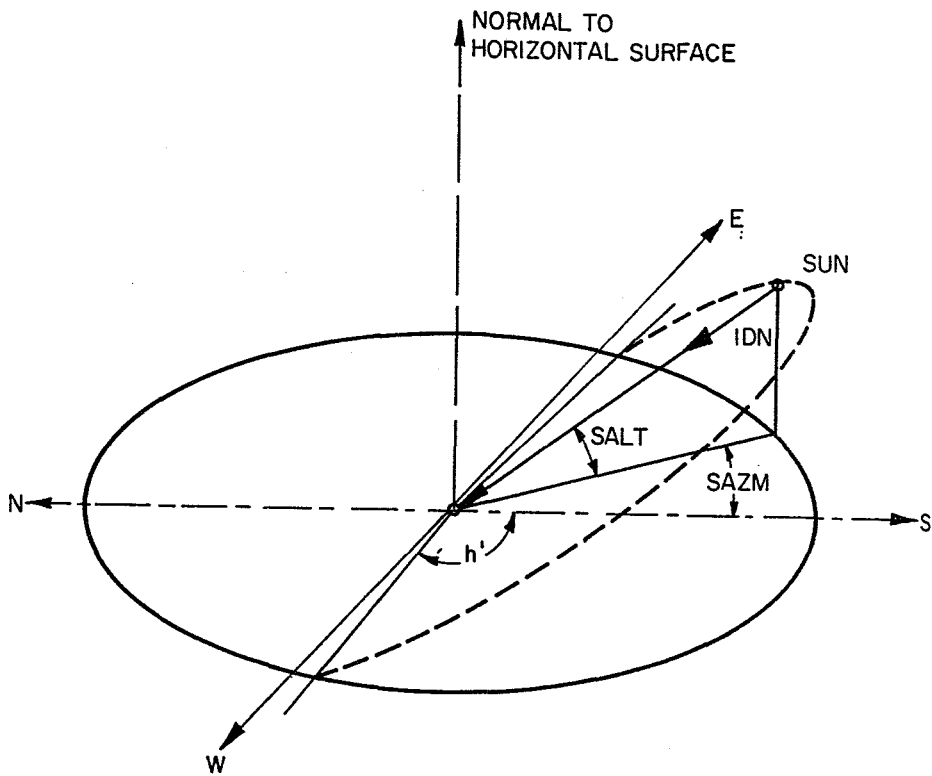


Figure A-2 Schematic Showing Apparent Path of Sun and Hour Angle

Figure A-3 Time Zones in the United States

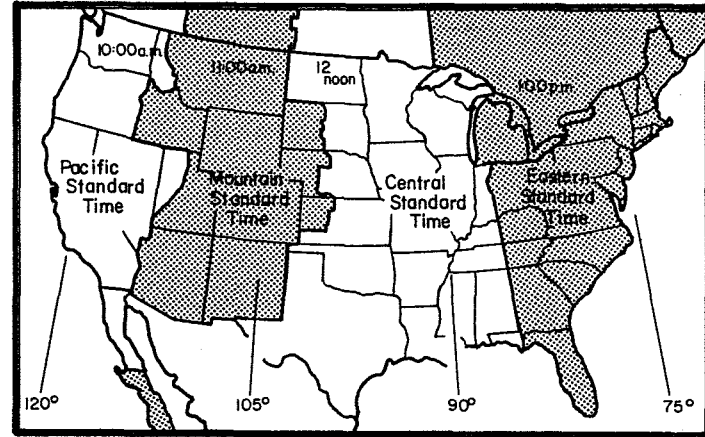


Table A-3 Time Zone Numbers in U. S. for Standard Time

TIME ZONE	TZN
Atlantic	4
Eastern	5
Central	6
Mountain	7
Pacific	8

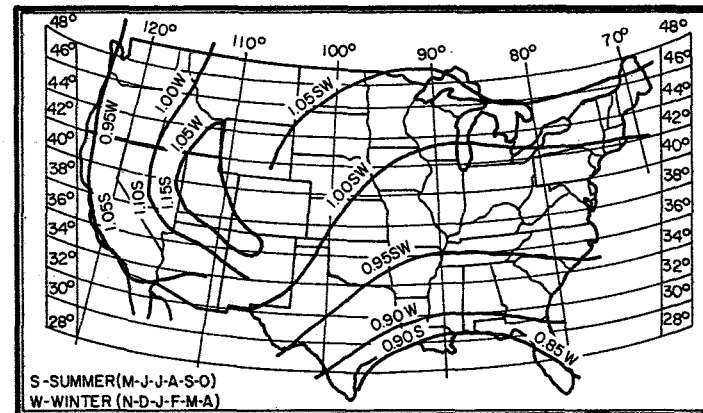


Figure A-4 Clearness Numbers of Non-Industrial Atmosphere in United States

Table A-4 lists, as function of date, five variables related to solar radiation. These variables are declination angle, δ ; the equation of time, ET; the apparent solar constant, A; the atmospheric extinction coefficient, B; the sky diffuse factor, C.

TABLE A-4 VALUES OF δ , ET, A, B AND C*

Date	δ Degrees	ET Hours	A Btu Per (hr) (sq ft)	B Air Mass ⁻¹	C
Jan. 21	-20.0	-.190	390	0.142	0.058
Feb. 21	-10.8	-.230	385	0.144	0.060
Mar. 21	0.0	-.123	376	0.156	0.071
Apr. 21	11.6	.020	360	0.180	0.097
May 21	20.0	.060	350	0.196	0.121
June 21	23.45	-.025	345	0.205	0.134
July 21	20.6	-.103	344	0.207	0.136
Aug. 21	12.3	-.051	351	0.201	0.122
Sept. 21	0.0	.113	365	0.177	0.092
Oct. 21	-10.5	.255	378	0.160	0.073
Nov. 21	-19.8	.235	387	0.149	0.063
Dec. 21	-23.45	.033	391	0.142	0.057

* Derived from the 1972 ASHRAE Handbook of Fundamentals, Table 1, p. 387, Chapter 22.

Calculation Sequence:

- Determine δ , ET, A, B, and C from Table A-4
- $h' = \cos^{-1} (-\tan(L) \cdot \tan(\delta))$ (see Figure A-2)
- $Y = h' \cdot 12 / \pi$
- Sunrise time (SRT) and sunset time (SST) in hr
 $SRT = 12 - Y - ET - TZN + l/15$
 $SST = 24 - SRT$
- Hour angle h in degrees
 $h = 15 \cdot (t - 12 + TZN + ET) - l$
 If $|h| > |h'|$ skip all the remaining calculations in this sequence and set
 $IDN = 0$
 $BS = 0$
 $BG = 0$
- Direction cosines of direct solar beam
 $\cos(Z) = \sin(L) \cdot \sin(\delta) + \cos(L) \cdot \cos(\delta) \cdot \cos(h)$
 $\cos(W) = \cos(\delta) \cdot \sin(h)$
 $\cos(S) = (1 - (\cos(Z))^{**2} - (\cos(W))^{**2})^{**0.5}$
 If $\cos(h) > \tan(\delta) / \tan(L)$, $\cos(S)$ is positive
- Solar altitude angle in radians
 $SALT = \sin^{-1} (\cos(Z))$
- Solar azimuth angle in radians
 $SAZM = \sin^{-1} (\cos(W) / \cos(SALT))$, if $\cos(S) > 0$
 $SAZM = \pi - \sin^{-1} (\cos(W) / \cos(SALT))$, if $\cos(S) < 0$
- Intensity of direct solar radiation for a cloudless condition
 $IDN = A \cdot CN \cdot \exp(-B / \cos(Z))$

10. Diffuse sky radiation (sky brightness) for a cloudless condition

$$BS = C * IDN / (CN) ** 2$$

11. Ground reflected radiation for a cloudless condition (ground brightness)

$$BG = \rho_g * (BS + IDN * \cos(Z))$$

Calculation Modification for Southern Hemisphere:

The preceding algorithm is applicable to the northern hemisphere only. For buildings in the southern hemisphere, the following modifications are required.

1. Shift values of B and C in Table A-4 by six months.

Values of δ , ET, A, B and C for the southern hemisphere are shown in Table A-5.

TABLE A-5 VALUES OF δ , ET, A, B and C FOR SOUTHERN HEMISPHERE

Date	δ Degrees	ET Hours	A Btu per (hr) (sq ft)	B Air ⁻¹ Mass	C
Jan. 21	-20.0	-.190	390	0.207	0.136
Feb. 21	-10.8	-.230	385	0.201	0.122
Mar. 21	0.0	-.123	376	0.177	0.092
Apr. 21	11.6	.020	360	0.160	0.073
May 21	20.0	.060	350	0.149	0.063
June 21	23.45	-.025	345	0.142	0.057
July 21	20.6	-.103	344	0.142	0.058
Aug. 21	12.3	-.051	351	0.144	0.060
Sept. 21	0.0	.113	365	0.156	0.071
Oct. 21	-10.5	.255	378	0.180	0.097
Nov. 21	-19.8	.235	387	0.196	0.121
Dec. 21	-23.45	.033	391	0.205	0.134

If $L \geq 0$ and if $\text{Cos}(h) > (\text{Tan}(\delta)/\text{Tan}(L))$, $\text{Cos}(s)$ is positive,
and if $\text{Cos}(h) \leq (\text{Tan}(\delta)/\text{Tan}(L))$, $\text{Cos}(s)$ is negative.

If $L < 0$ and if $\text{Cos}(h) \leq (\text{Tan}(\delta)/\text{Tan}(L))$, $\text{Cos}(s)$ is positive,
if $\text{Cos}(h) > (\text{Tan}(\delta)/\text{Tan}(L))$, $\text{Cos}(s)$ is negative.

CCF

An Algorithm for the Calculation of Cloudy Day Solar Radiation

This routine estimates the factor called CCF to modify the total solar radiation on a horizontal surface with the observed cloud cover data for a cloudy sky condition. The cloud cover observations are made every hour at major weather stations by experienced observers who estimate the amount of cloud on a scale of 0 to 10 and indicate the type of cloud in four different layers. Kimura and Stephenson^{1/} analyzed 1967 Canadian data for observed solar radiation with respect to the cloud cover data, type of cloud, and the calculated solar radiation under a cloudless condition at the same solar time. Based upon their analysis, a comprehensive methodology was developed for calculating the cloudy day solar radiation. The value of CCF, Cloud Cover Factor, is first defined as follows:

$$\text{CCF} = \text{ITHC}/\text{ITH}$$

where

ITHC: Total solar radiation on a horizontal surface
under a cloudy sky of given cloud amount and
types of cloud

ITH: Total solar radiation calculated for a horizontal surface under a cloudless sky at the same solar hour as of ITHC

Data:

- IS: Season index
- CA_j: Cloud amount at the j-th layer, where j = 1, 2, 3, and 4
- TOC_j: Type of cloud at the j-th layer, where j = 1, 2, 3, and 4
- TCA: Total cloud amount

Calculation Sequence:

1. $X = (\sum CA_j)_{\text{cirrus}} + (\sum CA_j)_{\text{cirrostratus}} + (\sum CA_j)_{\text{cirrocumulus}}$

2. Cloud cover

$CC = TCA - 0.5 \cdot X$

3. Cloud cover factor

$CCF = P + Q \cdot CC + R \cdot CC^2$

where P, Q, and R are found in the following table.

Table A-6

Season	P	Q	R
spring	1.06	0.012	-0.0084
summer	0.96	0.033	-0.0106
autumn	0.95	0.030	-0.0108
winter	1.14	0.003	-0.0082

The value of P, which is essentially the cloudless sky factor, depends upon the proportion of direct to diffuse sky radiation in reference to the standard ASHRAE values published in the 1972 Handbook of Fundamentals. If the value of P is unity, this proportion of direct to diffuse solar radiation is such that the solar radiation evaluated for a hori-

zontal surface under a cloudless sky should be equal to the value obtained by the method described in the 1972 ASHRAE Handbook of Fundamentals. If the value of P is different from unity, the direct to diffuse proportion is different from the standard values.

SOLAD

An Algorithm for Determining Diffuse and Direct Radiation Falling Onto a Surface

This routine determines the total as well as the diffuse and direct components of solar radiation incident on a given surface under either clear or cloudy sky by using the cloudless sky data calculated in the SUN routine and the cloud cover factor CCF calculated as described in the previous section.

Data:

- P: Cloudless sky factor shown in Table (A-6) in the CCF routine
- C: Standard diffuse sky factor shown in Table A-4 in the SUN routine
- CC: Cloud cover calculated in the CCF routine
- CCF: Cloud cover factor determined by the CCF routine
- WA: Azimuth angle of the surface under consideration in radians from south; + if west and - if east of south
- WT: Tilt angle of the surface under consideration in radians from the horizontal surface; zero for the horizontal surface and $\pi/2$ for the vertical walls

COS(Z), COS(W), AND COS(S):

Direction cosines of direct radiation (Calculated in SUN)

IDN: Intensity of the direct normal solar radiation for a cloudless condition in Btu per (hr) (sq ft) (Calculated in SUN)

BS and BG: Diffuse radiation from the cloudless sky and that from ground in Btu per (hr) (sq ft) (Calculated in SUN)

SALT: Solar altitude angle in radians (Calculated in SUN)

Calculation Sequence:

1. Let $X = \text{SIN}(SALT)$
2. $Y = 0.309 - 0.137 * X + 0.394 * X ** 2$
3. $K = X / (C + X) + (P - 1) / (1 - Y)$
4. Direct radiation on a horizontal surface under a cloudless sky
 $IDH = IDN * \text{COS}(Z)$
5. Diffuse radiation on a horizontal surface under a cloudless sky
 $IdH = BS$
6. Total radiation on a horizontal surface under a cloudless sky
 $ITH = IDH + IdH$
7. Direct radiation on a horizontal surface under a cloudy sky
 $IDHC = ITH * K * (1 - CC/10)$

8. Direction cosines of normal to the surface under consideration (the surface has an azimuth angle of WA and a tilt angle of WT)

$$\alpha = \cos(WT)$$

$$\beta = \sin(WA) * \sin(WT)$$

$$\gamma = \cos(WA) * \sin(WT)$$

9. Cosine of the incident radiation on the surface under consideration

$$\cos(\eta) = \alpha \cos(Z) + \beta \cos(W) + \gamma \cos(S)$$

10. Direct radiation on a surface under consideration under a cloudless sky

$$ID = IDN * \cos(\eta)$$

$$= 0 \text{ if } \cos(\eta) \leq 0$$

11. Direct radiation on a surface under a cloudy sky

$$IDC = ID * IDHC / IDH$$

12. Diffuse radiation for a cloudless sky

$$Id = BS \text{ for the horizontal surface}$$

$$Id = BS * Y + BG / 2 \text{ for the vertical surfaces*}$$

$$\text{where } Y = 0.55 + 0.437 * U + 0.313 * U ** 2$$

$$U = \cos(\eta)$$

$$\text{if } U \leq -0.2, Y = 0.45$$

13. Diffuse radiation upon a horizontal surface under a cloudy sky

$$IdHC = ITH * (CCF - K * (1 - CC / 10))$$

14. Diffuse radiation on a surface under consideration

$$IdC = Id * IdHC / IdH$$

15. Total radiation upon a surface under a cloudy sky

$$ITC = IDC + IdC$$

When the cloud cover CC is zero,

$$ITC = IT = ID + Id$$

* Diffuse radiation data for surfaces other than vertical and horizontal ones have not been analyzed sufficiently to date to provide a calculation procedure.

TAR

An Algorithm for Calculating Transmission, Absorption and Reflection Factors for Windows

Data:

$\text{Cos}(\eta)$: Cosine of angle of incidence of direct solar radiation (Calculated in SUN)

$k \cdot l$: Extinction coefficient [inches^{-1}] * thickness
[inches]

NOTE: In some cases, glass manufacturers provide the value of transmission at normal incidence. In this case, using the curve given in Figure A-5, it is possible to obtain the value of $k \cdot l$. The data for the curve are taken from reference 2.

Calculation Sequence:

A. Single-Pane Glass

1. Cosine of refraction angle

$$\text{COS}(\xi) = \text{SQRT} (1 - (1 - \text{COS}(\eta)) ** 2) / n$$

where $n = 1.520$, which is the index of refraction for ordinary glass

2. The fraction of radiation that is absorbed in a single pass through a sheet of glass of extinction coefficient $k \cdot l$

$$a = 1 - \text{Exp} (-k \cdot l / \text{COS}(\xi))$$

3. Single glass air-glass interface reflectivity by the Fresnel's formula

vibration in parallel to the plane of glass

$$r = (\text{TAN} (\eta - \xi)) ** 2 / \text{TAN} (\eta + \xi)$$

vibration in normal to the plane of glass

$$r' = (\text{SIN} (\eta - \xi)) ** 2 / (\text{SIN} (\eta + \xi)) ** 2$$

4. Absorptivity for direct radiation

$$A_{\eta} = 0.5 * (x + x')$$

where $x = a * (1 - r) * (1 + r * (1 - a)) / (1 - r * r * (1 - a) * (1 - a))$

$$x' = a * (1 - r') * (1 + r' * (1 - a)) / (1 - r' * r' * (1 - a) * (1 - a))$$

5. Transmissivity for direct radiation

$$T_{\eta} = 0.5 * (y + y')$$

where $y = (1 - r) * (1 - r) * (1 - a) / (1 - r * r * (1 - a) * (1 - a))$

$$y' = (1 - r') * (1 - r') * (1 - a) / (1 - r' * r' * (1 - a) * (1 - a))$$

6. Absorptivity and transmissivity for diffuse radiation

$$A_d = \int_0^{\pi/2} A_{\eta} \text{SIN} (2\eta) d\eta$$

$$T_d = \int_0^{\pi/2} T_{\eta} \text{SIN} (2\eta) d\eta$$

B. Double-Pane Glass

For the double-pane window, transmissivity and absorptivity for the outer and inner panes can be calculated separately first by using the single-pane procedure described above. Those calculated single glass properties can be designated here as follows:

A_1 : Absorptivity of inner pane for direct radiation

A_2 : Absorptivity of outer pane for direct radiation

T_1 : Transmissivity of inner pane for direct radiation

T_2 : Transmissivity of outer pane for direct radiation

A_{1d} : Absorptivity of inner pane for diffuse radiation

A_{2d} : Absorptivity of outer pane for diffuse radiation

T_{1d} : Transmissivity of inner pane for diffuse radiation

T_{2d} : Transmissivity of outer pane for diffuse radiation

1. Reflectivity of inner and outer panes

$$R_{1\eta} = 1 - A_{1\eta} - T_{1\eta}$$

$$R_{2\eta} = 1 - A_{2\eta} - T_{2\eta}$$

$$R_{1d} = 1 - A_{1d} - T_{1d}$$

$$R_{2d} = 1 - A_{2d} - T_{2d}$$

2. Absorptivity of the double-glazed system

a. Direct radiation

$$A_{\eta, \text{outer}} = A_{2\eta} * (1 + R_{1\eta} * T_{2\eta} / (1 - R_{1\eta} * R_{2\eta}))$$

$$A_{\eta, \text{inner}} = A_{1\eta} * T_{2\eta} / (1 - R_{1\eta} * R_{2\eta})$$

b. Diffuse radiation

$$A_{d, \text{outer}} = A_{2d} * (1 + R_{1d} * T_{2d} / (1 - R_{1d} * R_{2d}))$$

$$A_{d, \text{inner}} = A_{1d} * T_{2d} / (1 - R_{1d} * R_{2d})$$

3. Transmissivity of the double

a. Direct radiation

$$T_{\eta} = T_{1\eta} * T_{2\eta} / (1 - R_{1\eta} * R_{2\eta})$$

b. Diffuse radiation

$$T_d = T_{1d} * T_{2d} / (1 - R_{1d} * R_{2d})$$

Since the calculation of transmissivity and absorptivity are quite involved, they have been precalculated by Stephenson^{3/} for various values of COS (η) and expressed as polynomial functions of COS (η). The polynomial coefficients are shown in Table A-3 for single and double glazed windows and the equations are as follows:

Single-pane, direct radiation transmission

$$T_{\eta} = \sum_{j=0}^5 t_j * (\text{Cos } (\eta) ** j)$$

Single-pane, diffuse radiation transmission

$$T_d = 2 * \sum_{j=0}^5 t_j / (j + 2)$$

Polynomial representations of absorption factors for
and diffuse radiation.

Double-pane, direct radiation transmission

$$A_{\eta,outer} = \sum_{j=0}^5 a_{j,outer} * ((\cos(\eta))^{**j})$$

$$A_{\eta,inner} = \sum_{j=0}^5 a_{j,inner} * ((\cos(\eta))^{**j})$$

Double-pane, diffuse radiation transmission

$$A_{d,outer} = 2 * \sum_{j=0}^5 a_{j,outer} / (j + 2)$$

$$A_{d,inner} = 2 * \sum_{j=0}^5 a_{j,inner} / (j + 2)$$

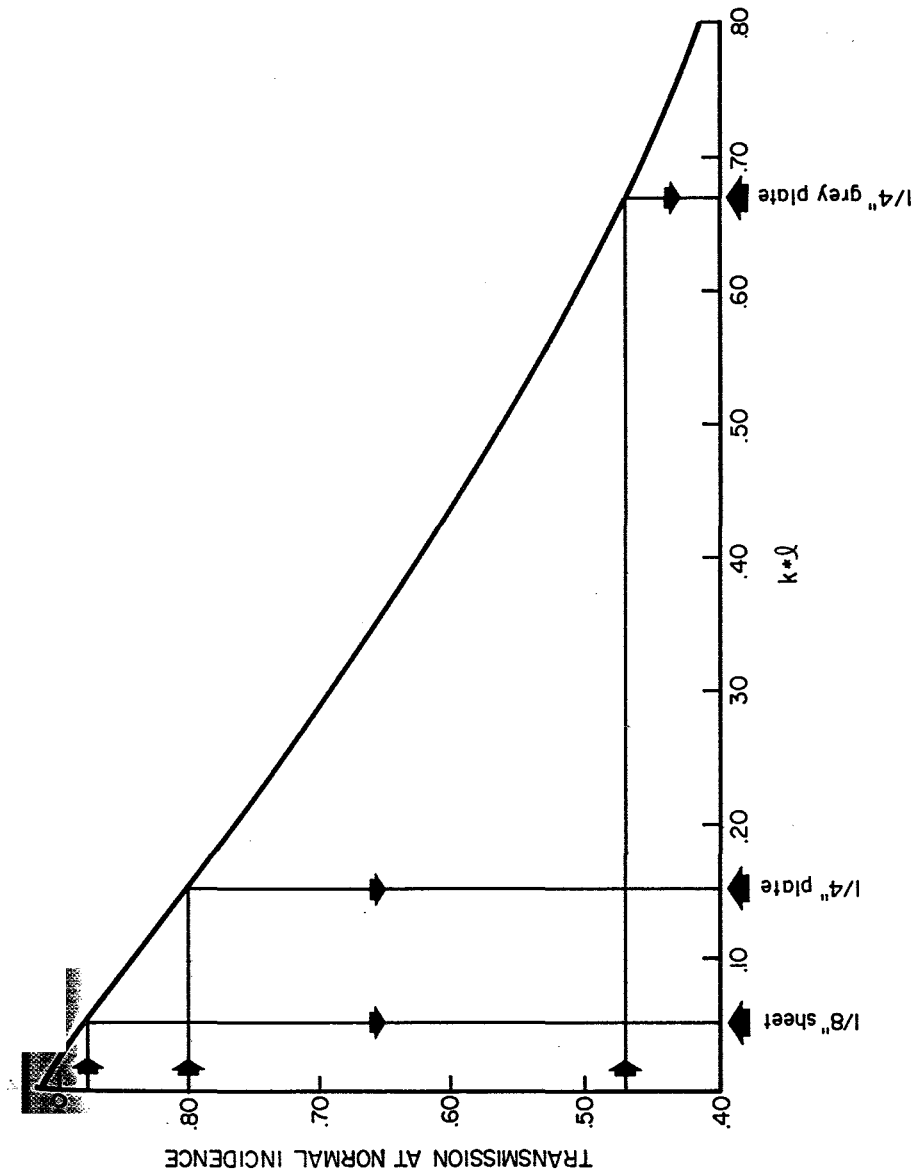


Figure A-5 Transmission at Normal Incidence Versus $k \cdot l$ for Single Sheet Glass

Table A-7 Polynomial Coefficients for Use in Calculation of Transmittance and Absorptance of Glass

SHG

An Algorithm for Calculating Solar Heat Gain Through Windows

k * u	j	Single Glazing		Double Glazing		
		a _j	t _j	a _{j,outer}	a _{j,inner}	t _j
0.05 1/8" Sheet	0	0.01154	-0.00865	0.01407	0.00228	-0.00401
	1	0.77674	2.71235	1.06226	0.34559	0.74050
	2	-3.94657	-0.62062	-5.59131	-1.19908	7.20350
	3	8.57881	-7.07329	12.15034	2.22366	-20.11763
	4	-8.38135	9.75995	-11.78092	-2.05287	19.68824
5	3.01188	-3.89922	4.20070	0.72376	-6.74585	
0.10	0	0.01636	-0.01114	0.01819	0.00123	-0.00438
	1	1.40783	2.39371	1.86277	0.29788	0.57818
	2	-6.79030	0.42978	-9.24831	-0.92256	7.42065
	3	14.37378	-8.98262	19.49443	1.58171	-20.26848
	4	-13.83357	11.51798	-18.56094	-1.40040	19.79706
5	4.92439	-4.52064	6.53940	0.48316	-6.79619	
0.15 1/4" Reg. Plate	0	0.01837	-0.01200	0.01905	0.00067	-0.00428
	1	1.92497	2.13036	2.47900	0.26017	0.45797
	2	-8.89134	1.13833	-11.74226	-0.72713	7.41367
	3	18.40197	-10.07925	24.14037	1.14950	-19.92004
	4	-17.48648	12.44161	-22.64299	-0.97138	19.40969
5	6.17544	-4.83285	7.89954	0.32705	-6.66603	
0.20	0	0.01902	-0.01218	0.01862	0.00035	-0.00401
	1	2.35417	1.90950	2.96400	0.22974	0.36698
	2	-10.47151	1.61391	-13.48701	-0.58381	7.27324
	3	21.24322	-10.64872	27.13020	0.84626	-19.29364
	4	-19.95978	12.83698	-25.11877	-0.67666	18.75408
5	6.99964	-4.95199	8.68895	0.22102	-6.43968	
0.40	0	0.01712	-0.01056	0.01423	-0.00009	-0.00279
	1	3.50839	1.29711	4.14384	0.15049	0.16468
	2	-13.86390	2.28615	-16.66709	-0.27590	6.17715
	3	26.34330	-10.37132	31.30484	0.25618	-15.84811
	4	-23.84846	11.95884	-27.81955	-0.12919	15.28302
5	8.17372	-4.54880	9.36959	0.02859	-5.23666	
0.60	0	0.01406	-0.00835	0.01056	-0.00016	-0.00192
	1	4.15958	0.92766	4.71447	0.10579	0.08180
	2	-15.06279	2.15721	-17.33454	-0.15035	4.94753
	3	27.18492	-8.71429	30.91781	0.06487	-12.43481
	4	-23.88518	9.87152	-26.63898	0.02759	11.92495
5	8.03650	-3.73328	8.79495	-0.02317	-4.07787	
0.80 50% Trans. H.A. Plate	0	0.01153	-0.00646	0.00819	-0.00015	-0.00136
	1	4.55946	0.68256	5.01768	0.07717	0.04419
	2	-15.43294	1.82449	-17.21228	-0.09059	3.87529
	3	26.70568	-6.95325	29.46388	0.00050	-9.59069
	4	-22.87993	7.80647	-24.76915	0.06711	9.16022
5	7.57795	-2.94454	8.05040	-0.03394	-3.12776	
1.00	0	0.00962	-0.00496	0.00670	-0.00012	-0.00098
	1	4.81911	0.51403	5.18781	0.05746	0.02576
	2	-15.47137	1.47607	-16.84820	-0.05878	3.00400
	3	25.86516	-5.41985	27.90292	-0.01855	-7.33834
	4	-21.69106	6.05546	-22.99619	0.06837	6.98747
5	7.08714	-2.28162	7.38140	-0.03191	-2.38328	

Data:

IDN: Intensity of direct normal solar radiation, Btu per (hr) (sq ft), (Calculated in SUN)

BS: Sky brightness, Btu per (hr) (sq ft), (Calculated in SUN)

BG: Ground brightness, Btu per (hr) (sq ft), (Calculated in SUN)

Cos(η): Cosine of the angle of incidence of direct solar radiation, (Calculated in SUN)

FWS: Form factor between the window and the sky*

FWG: Form factor between the window and the ground*

RO, RA, RI: Thermal resistances at outside surface, air space, and inside surface respectively, (sq ft) (hr) (F) per Btu

SLA: Sunlit area factor (Calculated in SHADOW)

SC: Shading coefficient if the window is shaded by drapes or blinds or if it has an inter-pane separation of more than 1 inch

* If more accurate data are not available, use FWS = FWG = 0.5.

NOTE: When the value of SC is given, these transmission and absorption factors should be for the standard 1/8" thick double strength glass (or $k^*g = 0.05$ of TAR) regardless of the type of glass used.

T_{η} , T_d : Transmission factors for direct and diffuse solar radiation for windows (Calculated in TAR)

$A_{\eta,outer}$, $A_{\eta,inner}$, $A_{d,outer}$, $A_{d,inner}$: Absorption factors for direct and diffuse solar radiation through outer and inner window panes (Calculated in TAR), respectively

Calculation Sequence:

1. Inward flowing fraction of the radiation absorbed by the inner and the outer pane, respectively

$$NI = (RO + RA)/(RO + RA + RI)$$

$$NO = RO/(RO + RA + RI)$$

2. Let

$$D = SIA * IDN * \cos(\eta) * (T_{\eta} + NO * A_{\eta,outer} + NI * A_{\eta,inner})$$

$$d = (BS * FWS + BG * FWG) * (T_d + NO * A_{d,outer} + NI * A_{d,inner})$$

3. Solar heat gain through window

If SC = 0, SHG = D + d

If SC ≠ 0, SHG = (SC) * (D + d) _{$k^*g=0.05$}

SHADOW

A Brief Description of the Procedures for Calculating External Shadows on a Building

A major portion of the air conditioning load on modern commercial buildings comes from solar radiation. To improve the accuracy of load assessment, it is necessary to know how much of a building is shaded and how much lies exposed to the sun's rays.

A new technique developed by Groth and Lokmanhekim^{4/} employs the representation of all architectural forms as a series of plane polygons. Even curved surfaces can be so represented. For example, a sphere may be approximated by the 20 sides of a regular icosahedron. This approximation gives a maximum error of only 3% in the shadow area cast by the sphere.

The output of the algorithm is not only the sunlit area, but also a pictorial display of the shadows and the surface upon which they are cast.

Coordinate Transformation:

Designate the polygons which cast shadows as shading polygons (SP) and those upon which shadows are cast as receiving polygons (RP). The vertex coordinates of each RP, and its relevant SP's, are transformed from a base coordinate system, xyz, to a new coordinate system, x'y'z', with origin 0 attached to the plane of the RP. The first three vertices,

V_1 , V_2 , and V_3 , of the RP being examined are used to define this new coordinate system. The x' axis passes through V_2 and V_3 , while the y' axis passes through V_1 . In order that the z' axis point outward from the surface, angle $V_1V_2V_3$ must be convex and the vertices must be numbered counterclockwise. The equation of transformation is written in matrix form as

$$\vec{x}' = A (\vec{x} - \vec{x}_0)$$

where

$$\vec{x}_0 = \vec{x}_2 + \gamma(\vec{x}_3 - \vec{x}_2)$$

$$\gamma, A \text{ Scaler} = (\vec{x}_1 - \vec{x}_2) \cdot (\vec{x}_3 - \vec{x}_2) / (\vec{x}_3 - \vec{x}_2) \cdot (\vec{x}_3 - \vec{x}_2)$$

$$\text{1st row of } A = (\vec{x}_3 - \vec{x}_0) / |\vec{x}_3 - \vec{x}_0|$$

$$\text{2nd row of } A = (\vec{x}_1 - \vec{x}_0) / |\vec{x}_1 - \vec{x}_0|$$

$$\text{3rd row of } A = \text{1st row of } A \times \text{2nd row of } A$$

Solar altitude, α , and azimuth, β , must also be transformed, into the solar direction vector, as

$$\vec{x}'_s = \begin{pmatrix} \sin\beta \cdot \cos\alpha \\ \sin\alpha \\ \cos\beta \cdot \cos\alpha \end{pmatrix}$$

Clipping Transformation:

Any part of an SP whose z' is negative cannot cast a shadow on the RP. These "submerged" portions of the SP's must be clipped off, prior to projection, lest they project "false" shadows (see Figure A-6). This

is done by finding, through linear interpolation, the points A and B, on the perimeter of the SP, which pierce the plane of the RP, and taking these points as new vertices. All submerged vertices are deleted. This results in a new polygon with line AB as a side, which will project only real shadows.

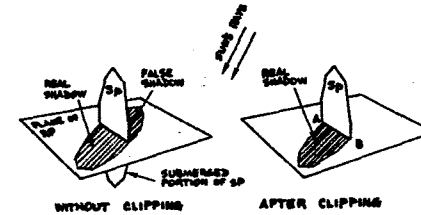


Figure A-6 Clipping

Projection Transformation:

To simulate the actual casting of a shadow, the following transformation projects, along the sun's rays, all the vertex points of the transformed and clipped RP's.

$$X = x' - \frac{x'_s}{z'_s} z'$$

$$Y = y' - \frac{y'_s}{z'_s} z'$$

Enclosure Test:

The coordinate, clipping and projection transformation have converted all RP and SP's in space into two dimensional figures in the RP plane. It remains only to find the points in the RP plane which lie inside the RP and inside one or more of the SP projections, i.e., points of the RP which are shaded. At this point, the two-space XY is divided into a grid and the center of each element of this grid is tested for enclosure by the RP and the SP projections. A point, P, whose coordinates are X_P, Y_P , is inside of polygon V_1, V_2, \dots, V_n if the following inequality holds.

$$\sum_{i=1}^n \Delta\theta_i \neq 0$$

The angular change, $\Delta\theta_i$, subtended at P by the ith side, and counted positive counterclockwise, is given by the following formulae.

$$\Delta\theta_i = \begin{cases} \theta_j - \theta_i & \text{if } |\theta_j - \theta_i| < 2 \\ \frac{(\theta_i - \theta_j)(4 - |\theta_i - \theta_j|)}{|\theta_j - \theta_i|} & \text{if } |\theta_j - \theta_i| \geq 2 \end{cases}$$

$$j = \begin{cases} i + 1 & \text{if } i < n \\ 1 & \text{if } i = n \end{cases}$$

$$\theta_i \sim \begin{matrix} \frac{Y_i - Y_P}{X_i - X_P + Y_i - Y_P} & \text{in 1st} & 1 + \frac{X_P - X_i}{X_P - X_i + Y_i - Y_P} & \text{in 2nd} \\ & \text{quadrant} & & \text{quadrant} \\ 2 + \frac{Y_P - Y_i}{X_P - X_i + Y_P - Y_i} & \text{in 3rd} & 3 + \frac{X_i - X_P}{X_i - X_P + Y_P - Y_i} & \text{in 4th} \\ & \text{quadrant} & & \text{quadrant} \end{matrix}$$

These approximate formulae, which express $\Delta\theta_i$ in right angles, replace the time-consuming square root and arc cosine computer library routines. They have, by set theory, been proved adequate for the purpose.

Display Matrix and Sample Problem:

An alphameric matrix is created corresponding to the grid elements in the RP plane. A blank component represents a grid element either outside the RP or exposed to the sun. An asterisk component represents a shaded grid element or one on the RP's boundary. Grid elements shaded by a transmissive structure are randomly asterisked with a probability equal to the fraction of incident light stopped by the shading structure. Figure A-7 shows the solution of a typical problem involving a transmissive structure.

SHADOW 1*

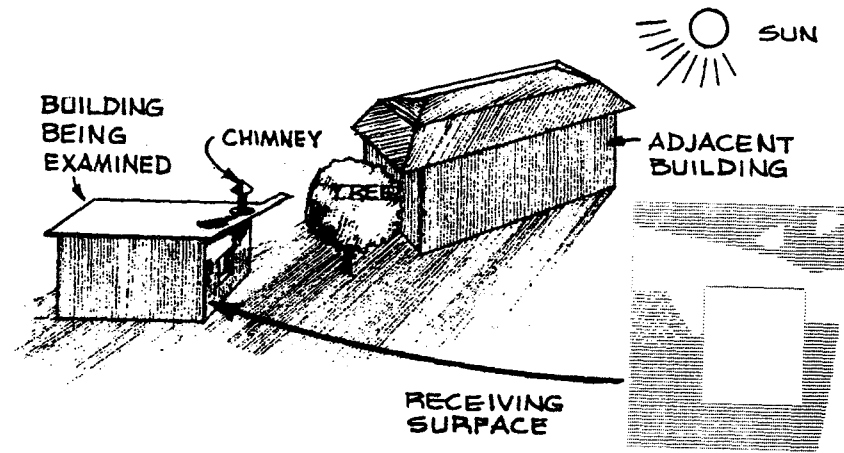


Figure A-7 The Computer Output of a Typical Problem

An Algorithm to Find the Ratio of the Sunlit and Shaded Area of a Given Window Where the Shadows are Cast by Various Combinations of Overhang and Side Fins

Data: (Variable names corresponding to the FORTRAN listing included in this section. Right and left is determined facing the window from outside - see Figure A-8).

HT: Window height

FL: Window width

FP: Depth of the overhang

AW: Distance from top of the window to the overhang

BWL: Distance of the overhang extended beyond the left edge of the window

BWR: Distance of the overhang extended beyond the right edge of the window

D: Depth of vertical projection at the end of the overhang

FP1: Depth of the left fin

A1: Distance of the left fin extended above the top of the window

B1: Distance from the left edge of the window to the left fin

C1: Distance of the left fin stop short above the bottom of the window

FP2: Depth of the right fin

* This section was contributed by Tseng-Yao Sun; Ayres and Hayakawa, Los Angeles, California.

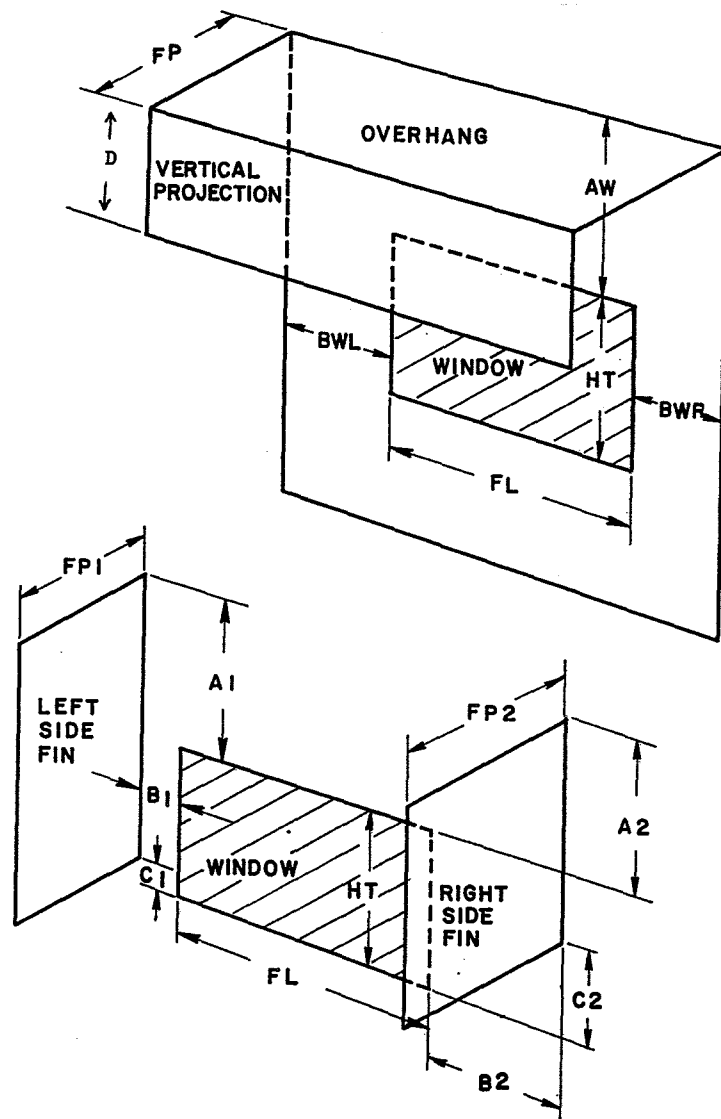


Figure A-8 Shadow 1 Input Data

A2: Distance of the right fin extended above the top of the window

B2: Distance from the right edge of the window to the right fin

C2: Distance of the right fin stop short above the bottom of the window

PHI: Solar azimuth angle

WAZI: Window azimuth angle

COSZ: Cosine of solar zenith angle

Calculation Sequence:

The principle calculation sequence of this subroutine is described in reference 5 and the principle output of the subroutine is the variable SHRAT--the shade ratio or ratio of sunlit area to the total window area. The treatment of shadow overlapping cast by various shading devices is not discussed in the reference but is included in the FORTRAN listing.

```

SUBROUTINE SHADOW(SHDX,PHI,COSZ,SHRAT)
DIMENSION SHDX(20)
FI=SHDX(1)
HI=SHDX(2)
FP=SHDX(3)
AW=SHDX(4)
BWL=SHDX(5)
BWR=SHDX(6)
D=SHDX(7)
FP1=SHDX(8)
A1=SHDX(9)
B1=SHDX(10)
C1=SHDX(11)
FP2=SHDX(12)
A2=SHDX(13)
B2=SHDX(14)
C2=SHDX(15)
WAZI=SHDX(16)
% THIS PROGRAM CALCULATES SHADOW CAST BY OVERHANG AND SIDE FINS
% PHI....SOLAR AZIMUTH ANGLE
% COSZ...COSINE OF SOLAR ZENITH ANGLE
% SHRAT..SHADE RATIO:RATIO OF THE SUNLIT AREA TO THE TOTAL WINDOW AREA
% HT.....WINDOW HEIGHT
% FL.....WINDOW WIDTH
% FP.....DEPTH OF THE OVERHUNG
% AW.....DISTANCE FROM TOP OF THE WINDOW TO THE OVERHUNG
% BWL....DISTANCE OF THE OVERHUNG EXTENDED BEYOND THE LEFT EDGE OF THE WINDOW
% BWR....DISTANCE OF THE OVERHUNG EXTENDED BEYOND THE RIGHT EDGE OF THE WINDOW
% D.....DEPTH OF VERTICAL PROJECTION AT THE END OF THE OVERHUNG
% FP1....DEPTH OF THE LEFT FIN
% A1.....DISTANCE OF THE LEFT FIN EXTENDED ABOVE THE TOP OF THE WINDOW
% B1.....DISTANCE FROM THE LEFT EDGE OF THE WINDOW TO THE LEFT FIN
% C1.....DISTANCE OF THE LEFT FIN STOP SHORT ABOVE THE BOTTOM OF THE WINDOW
% FP2....DEPTH OF THE RIGHT FIN
% A2.....DISTANCE OF THE RIGHT FIN EXTENDED ABOVE THE TOP OF THE WINDOW
% B2.....DISTANCE FROM THE RIGHT EDGE OF THE WINDOW TO THE RIGHT FIN
% C2.....DISTANCE OF THE RIGHT FIN STOP SHORT ABOVE THE BOTTOM OF THE WINDOW
% WAZI...WINDOW AZIMUTH ANGLE
% SHRAT=1.
1103 A=AW
      H=HT
      GAMMA=PHI-WAZI
      COSG=COS(GAMMA)
      IF(COSG)100,100,104
100  SHRAT=0.
      GO TO 2000
104  CONTINUE
      SBETA=COSZ
      IF(SBETA)100,100,152
152  SING=SIN(GAMMA)
      VERT=SBETA/SQRT(1.-SBETA*SBETA)/COSG
      HORIZ=ARS(SING)/COSG
      TCETA=VERT/HORIZ

```

```

IF(GAMMA) 155,154,154
% -----SUN ON LEFT
154 B=BWL
      GO TO 156
% -----SUN ON RIGHT
155 B=BWR
156 ARSHF=0.
      AREAV=0.
      ARSIF=0.
      AREA0=0.
      AREA1=0.
      ARSH1=0.
      FL3=0.
      H3=0.
      H1=H
      FL1=FL
      K=1
      L=1
      T1=FP*VERT
      FM1=FP*HORIZ
      IF(FP)37,37,153
153 T=T1
      FM=FM1
      AB=B*TCETA
      UG=(FL+B)*TCETA
      DE=(H+A)/TCETA
% -----HORIZONTAL OVERHUNG "AREA0"
      IF(T-A)27,27,2
      IF(AB-A)14,14,3
      IF(DE-B)12,12,4
      IF(FM-B)11,11,5
      IF(DE-(FL+B))8,8,6
      IF(FM-(FL+B))9,7,7
% -----HORIZ 9
      AREA0=FL*(0.5*(AB+UG)-A)
      GO TO 37
      IF(T-(H+A))9,10,10
% -----HORIZ 7
      AREA0=(T-A)*FL-((FM-B)**2)*TCETA*0.5
      L=2
      GO TO 21
% -----HORIZ 8
10  AREA0=H*FL-(DE-B)**2*TCETA*0.5
      GO TO 37
% -----HORIZ 3
11  AREA0=FL*(T-A)
      L=2
      GO TO 24
12  IF(T-(H+A))11,13,13
% -----HORIZ 2
13  AREA0=H*FL
      GO TO 68
14  IF(UG-A)27,27,15
15  IF(DE-(FL+B))18,18,16

```

```

16 IF (FM-(FL+B)) 20,17,17
% -----HORIZ 6
17 AREA0=(UG-A)**2/TCETA*0.5
GO TO 37
18 IF (T-(H+A)) 20,19,19
% -----HORIZ 5
19 AREA0=H*(FL-(A+0.5*H)/TCETA+B)
GO TO 37
% -----HORIZ 4
20 AREA0=(T-A)*(FL+B-FM*(1.+A/T)*0.5)
L=2
% -----VERT PROJ "AREAV"
21 FL3=FL+B-FM
IF (T+D-(H+A)) 22,22,23
% -----VERT 8
22 H3=D
GO TO 3700
% -----VERT 9
23 H3=H+A-T
GO TO 3700
24 FL3=FL
IF (T+D-(H+A)) 26,26,25
% -----VERT 7
25 H3=H+A-T
AREAV=H3*FL3
GO TO 68
% -----VERT 6
26 H3=D
GO TO 3700
27 IF (T+D-A) 37,37,28
28 IF (FM-B) 34,34,29
29 IF (FM-(FL+B)) 31,37,37
31 FL3=FL+B-FM
IF (T+D-(H+A)) 33,33,32
% -----VERT 5
32 H3=H
GO TO 3700
% -----VERT 4
33 H3=T+D-A
GO TO 3700
34 IF (T+D-(H+A)) 36,35,35
% -----VERT 2
35 AREAV=H*FL
GO TO 68
% VERT 3
36 H3=T+D-A
FL3=FL
3700 AREAV=FL3*H3
% -----SIDE FIN AND SHORT SIDE FIN
% -----SIDE FIN "AREA1" "ARSIF"
37 IF (GAMMA) 66,68,74
74 FPF=FP1
AF=A1
BF=B1

```

```

CX=C1
GO TO 84
66 FPF=FP2
AF=A2
BF=B2
CX=C2
84 IF (FPF) 68,68,67
67 T=FPF*VERT
FM=FPF*HORIZ
AF1=AF
IF (AREA0) 73,73,88
% -----TEST FOR OVERLAP OF FIN AND OVERHUNG SHADOW
88 AT=A+(BF-B)*TCETA
IF (AT-AF) 711,73,73
% -----OVERLAP EXISTS..L=2 IF OVERHUNG SHADOW HAS HORIZ EDGE IN WINDOW
711 GO TO (621,712),L
% -----TEST FOR TYPE OF OVERLAP
712 IF ((FM-BF)-(FM1-B)) 621,622,622
% -----SET L=1,SHADOW INTERSECT ON INCLINED EDGE OF OVERHUNG SHADOW
% -----FIN SHADOW IS BELOW INCLINED EDGE OF OVERHUNG SHADOW
621 AF=AT
L=1
GO TO 73
% -----L IS 2, HORIZ EDGE OF OVERHUNG SHADOW--PORTION ABOVE HORIZ EDGE
% -----NOT IN OVERHUNG SHADOW IS FIN SHADOW
622 AREA1=FL*(T1-A)-AREA0
% -----RESET TO CALC FIN SHADOW BELOW HORIZ EDGE OF OVHNG SHADOW
AF=T1-A+AF1
H=H+AF1-AF
% -----SHADOW OF FIN (K=1 ON GLASS K=2 ON VERT PROJ SHADOW)
73 AB=BF*TCETA
UG=(FL+BF)*TCETA
DE=(H+AF)/TCETA
DJ=CX/TCETA
IF (FM-BF) 69,69,38
38 IF (AB-AF) 39,50,50
39 IF (UG-AF) 48,48,40
40 IF (T-AF) 47,47,41
41 IF (UG-(H+AF)) 44,44,42
42 IF (T-(H+AF)) 91,80,80
% -----FIN 9
80 AREA1=H*((AF+H*0.5)/TCETA-BF)+AREA1
GO TO 58
44 IF (FM-(FL+BF)) 91,89,89
% -----FIN 8
89 AREA1=H*FL-(UG-AF)**2/TCETA*0.5+AREA1
GO TO 58
% -----FIN 7
91 AREA1=(FM-BF)*H-(T-AF)**2/TCETA*0.5+AREA1
GO TO 63
48 IF (FM-(FL+BF)) 47,47,49
% -----FIN 3
47 AREA1=H*(FM-BF)+AREA1
GO TO 63

```

```

% -----FIN 2
49 AREA1=H*FL+AREA1
   GO TO 58
50 IF (DE-BF) 69,69,51
51 IF (UG-(H+AF)) 55,55,52
52 IF (T-(H+AF)) 93,94,94
% -----FIN 6
94 AREA1=(DE-BF)**2*TCETA*0.5+AREA1
   GO TO 58
% -----FIN 4
93 AREA1=(FM-BF)*(H+AF-(T+AB)*0.5)+AREA1
   GO TO 63
55 IF (FM-(FL+BF)) 93,99,99
% -----FIN 5
99 AREA1=FL*(H-(BF+FL*0.5)*TCETA+AF)+AREA1
% -----SHORT SIDE FIN "ARSH1","ARSHF"
58 IF (DJ-BF) 69,69,59
59 IF (DJ-(FL+BF)) 61,61,60
% -----SHORT 3
60 ARSH1=-FL*(CX-(BF+FL/2.)*TCETA)
   GO TO 69
% -----SHORT 4
61 ARSH1=-(CX-AB)**2/TCETA*0.5
   GO TO 69
63 IF (DJ-BF) 69,69,64
64 IF (DJ-FM) 61,61,65
% -----SHORT 2
65 ARSH1=-(FM-BF)*(CX-(T+AB)*0.5)
69 GO TO (77,76),K
76 ARSH1=-ARSH1
   AREA1=-AREA1
77 ARSHF=ARSHF+ARSH1
   ARSIF=ARSIF+AREA1
   GO TO (78,68),K
78 IF (AREAV) 68,68,72
% -----RESET PARAMETERS TO DEDUCT FIN SHADOW OVERLAP ON VERT PROJ SHADOW
72 K=2
   AREA1=0.
   ARSH1=0.
   BBF=BF
   BF=FM1-B+BF
   IF (BF) 186,185,185
186 BF=BBF
185 IF (HT+A-T1-D) 87,87,188
188 CX=CX-(HT+A-T1-D)
   IF (CX) 85,87,87
85 CX=0.
87 AF=T1-A+AF
   H=H3
   FL=FL3
   GO TO 73
% ----- SHADED AREA "ARSHA"
68 ARSHA=AREA0+AREAV+ARSHF+ARSIF
   SHRAT=(FL1*H1-ARSHA)/(FL1*H1)

```

```

2000 FL=FL1
      CONTINUE
      RETURN
      END

```

SHADOW 2*

An Algorithm to Determine Whether or Not a Given Window is Shaded by a Remote Object Such as an Adjacent Building

This algorithm is approximate and is applicable only where the window is relatively small in comparison to the shading object. Large windows may be subdivided into smaller segments for this consideration. The window is considered either completely shaded or completely in sun. Partially shaded window can be considered in either case depending on the location of the window reference point. Figure A-9 shows a typical window-shading object relationship.

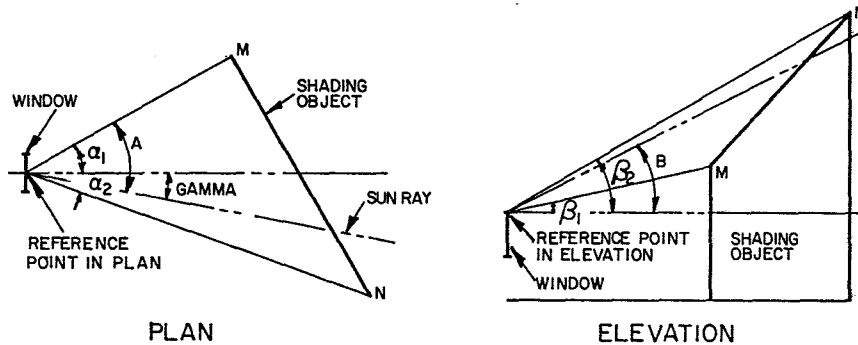


Figure A-9 A Typical Window-Shading Object Relationship

Note that the reference point can be located at any point on the window. Locating the reference point at the top of the window as shown in the elevation in Figure A-9 is slightly conservative as compared to if the reference point is located at the center of the window.

* This section was contributed by Tseng-Yao Sun; Ayres, Cohen and Hayakawa, Los Angeles, California.

Data:

- α_1, α_2 : Azimuth shadow limit angles. Right +, Left -
- β_1, β_2 : Altitude shadow limit angles
- WAZI: Window azimuth angle
- PHI: Solar azimuth angle
- BETA: Solar altitude angle

Calculation Sequence:

This subroutine determines whether the window is sunlit or shaded for the given position of the sun.

1. Wall-solar azimuth angle

$$\text{GAMMA} = \text{PHI} - \text{WAZI}$$

2. If $\text{GAMMA} < \alpha_1$ or $\text{GAMMA} > \alpha_2$, the window is in sun

3. If $\alpha_2 > \text{GAMMA} > \alpha_1$,

$$A = \text{GAMMA} - \alpha_1$$

$$B = \beta_1 + A * (\beta_2 - \beta_1)$$

4. If $\text{BETA} > B$, the window is in sun. Otherwise, the window is in shade.

FIJ*

An Algorithm for Calculating Radiation Shape Factors
Between Inside Surfaces of a Room

Definition of Room

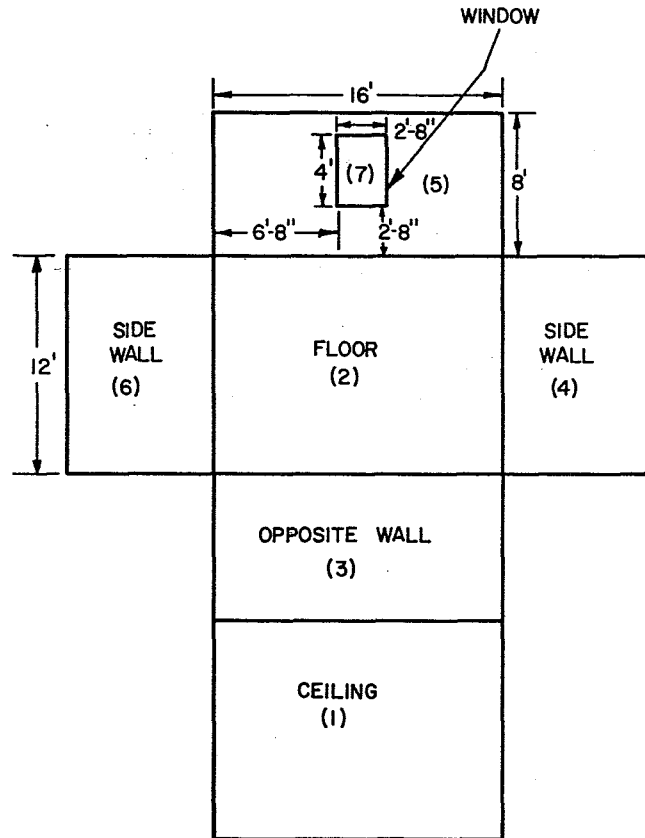


Figure A-10 Room Layout

Data:

- L: Length of room
- W: Width of room
- H: Height of room
- A: Height of windows or doors
- B: Width of windows or doors
- C: Distance of left edge of window from left wall
- D: Height of lower edge of window from floor

The primary variables determined by this subroutine are:

F_{m-n} : An array giving radiation shape factors between the various inside surfaces of a room

- m,n = 1 Ceiling
- 2 Floor
- 3 Wall No. 1 (length by height)
- 4 Wall No. 2 (side wall)
- 5 Wall No. 3 (opposite wall)
- 6 Wall No. 4 (side wall)
- 7-14 Provision for windows and doors in walls

* This section was contributed by D. M. Burch and B. A. Peavy; Thermal Engineering Section, Center for Building Technology, National Bureau of Standards, Washington, D.C.

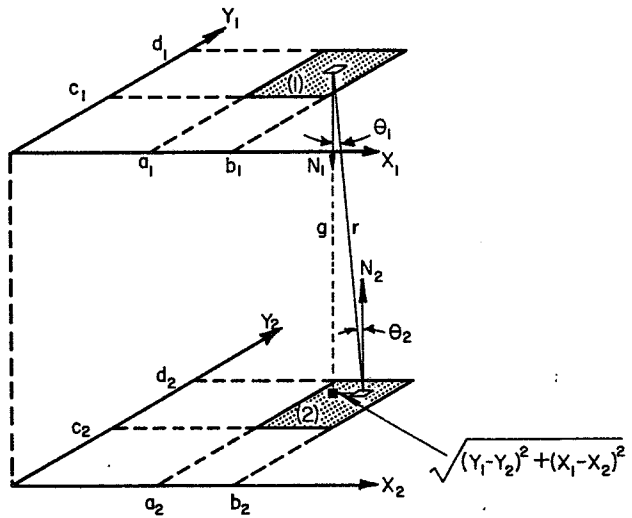


Figure A-11 Radiation Heat Exchange Between Ceiling and Floor Surfaces

Radiation shape factor, F_{1-2} , between two parallel room surfaces

$$2\pi(b_1 - a_1)(d_1 - c_1)F_{1-2} = [P(b_2 - b_1) + P(a_2 - a_1)] [Q(c_2 - c_1) + Q(d_2 - d_1) - Q(c_2 - d_1) - Q(d_2 - c_1)] + [P(b_2 - a_1) + P(a_2 - b_1)] [Q(c_2 - d_1) + Q(d_2 - c_1) - Q(c_2 - c_1) - Q(d_2 - d_1)]$$

where

$$P(Z_1)Q(Z_2) = Z_1 W \tan^{-1} \frac{Z_1}{W} + Z_2 V \tan^{-1} \frac{Z_2}{V} - \frac{g}{2} \ln \frac{(W^2 + Z_1^2)}{V^2}$$

$$V^2 = g^2 + Z_1^2, W^2 = g^2 + Z_2^2$$

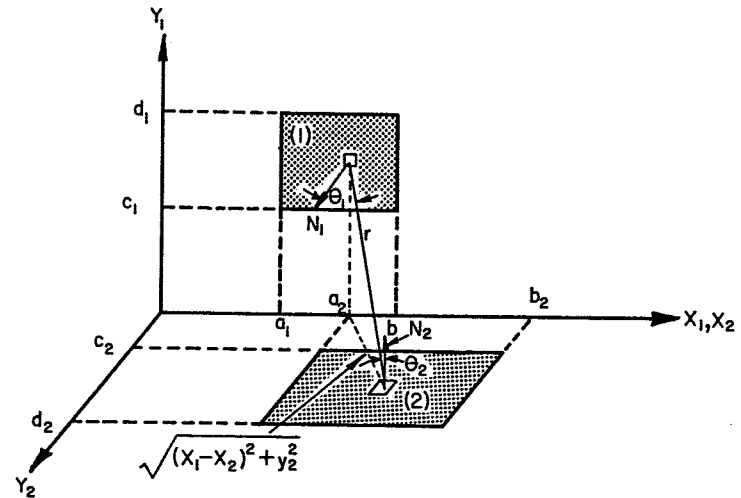


Figure A-12 Radiation Heat Exchange Between Wall and Floor Surfaces

Radiation Shape Factor, F_{1-2} , between two perpendicular room surfaces

faces

$$2\pi(b_1 - a_1)(d_1 - c_1)F_{1-2} = [R(b_2 - b_1) + R(a_2 - a_1)] [S(c_2 + c_1) + S(d_2 + d_1) - S(c_2 + d_1) - S(d_2 + c_1)] + [R(b_2 - a_1) + R(a_2 - b_1)] [S(c_2 + d_1) + S(d_2 + c_1) - S(c_2 + c_1) - S(d_2 + d_1)]$$

where

$$R(Z_1)S(Y_2 + Y_1) = TZ_1 \tan^{-1} \frac{Z_1}{T} + \frac{1}{4} (Z_1^2 - T^2) \ln \frac{(T^2 + Z_1^2)}{T^2}$$

$$T^2 = Y_2^2 + Y_1^2$$

Calculation Sequence:

1. Determine areas of ceiling, floor, and walls (no windows or doors),

$$A_m, m = 1, 2, 3, 4, 5, 6$$

2. Calculate radiation shape factor F_{m-n} for these surfaces using equations, and reciprocal relation

$$F_{m-n} = \frac{A_n}{A_m} F_{n-m}$$

3. Determine area of windows and doors and subtract from pertinent wall areas to give net wall areas
4. Calculate radiation shape factors from windows and/or doors to ceiling and/or floor using the above shape factor equations and the reciprocal relation. The radiation shape factors from a ceiling or floor surface to a window, door, or a wall area is given by

$$F_{m-n_k} = F_{m-n} - F_{m-n_1} - F_{m-n_2}$$

where k denotes the surface which is applicable for a receiving surface, A_n that has been subdivided into 2 or more surfaces.

5. Calculate radiation shape factors from windows and doors to walls using equations, above defined angle factor algebra and

$$A_k F_{m_k-n} = A_m F_{m-n} - A_{m_1} F_{m_1-n} - A_{m_2} F_{m_2-n}$$

which is applicable for a transmitting surface A_m that has been subdivided into 2 or more surfaces.

6. The resulting array F_{m-n} must be satisfied by the identity

$$\sum_{k=1}^p F_{m-k} = 1$$

where p is the number of surfaces visible to the transmitting surface A_m

C THIS PROGRAM DETERMINES THE RADIATION SHAPE FACTORS FOR A ROOM
 C OF ARBITRARY DIMENSIONS WITH THE PROVISION FOR T WINDOWS OR
 C DOORS OF ANY SIZE AND POSITION ON EACH OF THE FOUR WALLS
 C INPUT TO THE PROGRAM IS READ IN FIELDS OF NINE AND FIRST CARD IS

C LENGTH WIDTH HEIGHT

C THIS CARD IS FOLLOWED BY FOUR CARDS GIVING PERTINENT DIMENSIONS
 C FOR WINDOWS AND DOORS - FIRST CARD IS FOR WINDOWS OR DOORS ON A
 C WALL DEFINED ON THE LENGTH OF ROOM - SECOND CARD ON WIDTH, ETC.
 C LEAVE SPACES BLANK IF THERE IS NO WINDOW OR DOOR

A	B	C	D	A	B	C	D
A	B	C	D	A	B	C	D
A	B	C	D	A	B	C	D
A	B	C	D	A	B	C	D

C WHERE A=HEIGHT OF WINDOW, C=CORNER OF WALL TO LEFT EDGE OF WINDOW
 C B=WIDTH OF WINDOW, D=HEIGHT FROM FLOOR TO LOWER EDGE OF WINDOW
 C OUTPUT CONSISTS OF ASSIGNMENT OF NUMBERS TO THE VARIOUS SURFACES
 C OF THE ROOM

1	CEILING	LENGTH X WIDTH
2	FLOOR	LENGTH X WIDTH
3	WALL NO.1	LENGTH X HEIGHT
4	WALL NO.2	WIDTH X HEIGHT
5	WALL NO.3	LENGTH X HEIGHT
6	WALL NO.4	WIDTH X HEIGHT
7-14	WINDOW OR DOOR ON WALL NO.X	

C THIS IS FOLLOWED BY A PRINTOUT OF AN ARRAY OF RADIATION SHAPE FACTORS
 C DIMENSION SF(14,14),A(8),B(8),C(8),D(8),S(14),X(5)
 C DIMENSION IFIL(14),VF(14,14)

C REAL L
 C READ(5,1)L,W,H
 C READ(5,1)(B(I),D(I),A(I),C(I),I=1,8)
 C FORMAT(8F9.2)
 C 1 15 ON THE ROOM LENGTH/7HOM OR N/4H 1,6X,10H CEILING ,F6.2,4H BY
 C 2,F6.2/4H 2,6X,10H FLOOR ,F6.2,4H BY ,F6.2/4H 3,6X,10H WALL
 C 3NO.1,F6.2,4H BY ,F6.2/4H 4,6X,10H WALL NO.2,F6.2,4H BY ,F6.2/4H
 C 4 5,6X,10H WALL NO.3,F6.2,4H BY ,F6.2/4H 6,6X,10H WALL NO.4,F6.2
 C 5,4H BY ,F6.2)
 C 4,3 FORMAT(I4,6X,15H DOOR ON WALL,I2,4F11.2)
 C 3,2 FORMAT(I4,6X,15H WINDOW ON WALL,I2,4F11.2)
 C 3,3 FORMAT(6X,17H WINDOWS OR DOORS,13X,2H A,9X,2H B,9X,2H C,9X,2H D)
 C 3,4 FORMAT(1H /8X,18H A - WINDOW HEIGHT/8X,17H B - WINDOW WIDTH/9X,4AH
 C 1C - DISTANCE OF LEFT EDGE OF WINDOW TO LEFT WALL/8X,46H D - HEIGHT
 C 2 FROM FLOOR TO LOWER EDGE OF WINDOW)
 C WRITE (6,301) L,W,L,W,L,H,W,H,L,H,W,H
 C WRITE (6,303)
 C K=7
 C DO 305 I=1,8
 C IF(D(I).LT.1.E-8) GO TO 305
 C J=(I+1)/2
 C IC=C(I)*100.
 C IF(IC.EQ.0)GO TO 401
 C WRITE(6,302)K,J,D(I),B(I),A(I),C(I)
 C GO TO 402
 C 4,1 WRITE(6,403)K,J,D(I),B(I),A(I),C(I)
 C 4,2 CONTINUE
 C K=K+1

305 CONTINUE
 WRITE(6,304)
 DO 26 I=1,8
 B(I)=A(I)+B(I)
 26 D(I)=C(I)+D(I)
 S(1)=W*L
 S(2)=S(1)
 DO 15 I=1,8
 S(I+6)=(B(I)-A(I))*(D(I)-C(I))
 IA=S(I+6)
 IF(IA.EQ.0)S(I+6)=1.0E-08
 15 CONTINUE
 DO 14 I=1,5
 VAR=L
 IF(I.EQ.2.OR.I.EQ.4)VAR=W
 14 X(I)=VAR
 DO 25 I=1,4
 IW=(I-1)*2+7
 25 S(I+2)=X(I)*H
 DO 2 J=1,14
 2 SF(I,J)=0.0
 SF(1,2)=PF(0.0,L,0.0,W,0.0,L,0.0,W,H)
 SF(1,3)=AF(0.0,L,0.0,W,0.0,L,0.0,H)
 SF(1,4)=AF(0.0,W,0.0,L,0.0,W,0.0,H)
 SF(1,5)=SF(1,3)
 SF(1,6)=SF(1,4)
 SF(2,1)=SF(1,2)*S(1)/S(2)
 SF(2,3)=SF(1,3)
 SF(2,4)=SF(1,4)
 SF(2,5)=SF(2,3)
 SF(2,6)=SF(2,4)
 SF(3,1)=SF(1,3)*S(1)/S(3)
 SF(3,2)=SF(2,3)*S(2)/S(3)
 SF(3,4)=AF(0.0,H,0.0,L,0.0,H,0.0,W)
 SF(3,5)=PF(0.0,L,0.0,H,0.0,L,0.0,H,W)
 SF(3,6)=SF(3,4)
 SF(4,1)=SF(1,4)*S(1)/S(4)
 SF(4,2)=SF(2,4)*S(2)/S(4)
 SF(4,3)=SF(3,4)*S(3)/S(4)
 SF(4,5)=SF(4,3)
 SF(4,6)=PF(0.0,W,0.0,H,0.0,W,0.0,H,L)
 SF(5,1)=SF(1,5)*S(1)/S(5)
 SF(5,2)=SF(2,5)*S(2)/S(5)
 SF(5,3)=SF(3,5)*S(3)/S(5)
 SF(5,4)=SF(4,5)*S(4)/S(5)
 SF(5,6)=SF(5,4)
 SF(6,1)=SF(1,6)*S(1)/S(6)
 SF(6,2)=SF(2,6)*S(2)/S(6)
 SF(6,3)=SF(3,6)*S(3)/S(6)
 SF(6,4)=SF(4,6)*S(4)/S(6)
 SF(6,5)=SF(5,6)*S(5)/S(6)
 DO 250 I=1,4
 IW=(I-1)*2+7
 250 S(I+2)=S(I+2)-S(IW)-S(IW+1)
 DO 3 K=1,4
 J=(K-1)*2
 DO 5 I=1,2
 N=I+J
 N6=N+6
 SF(N6,2)=AF(A(N),B(N),C(N),D(N),0.0,X(K),0.0,X(K+1))
 SF(N6,1)=AF(A(N),B(N),H-D(N),H-C(N),0.0,X(K),0.0,X(K+1))

```

5 SF(2,N6)=SF(N6,2)*S(N6)/S(2)
  SF(1,N6)=SF(N6,1)*S(N6)/S(1)
  SF(2,2+K)=SF(2,2+K)-SF(2,J+7)-SF(2,J+8)
  SF(1,2+K)=SF(1,2+K)-SF(1,J+7)-SF(1,J+8)
3 SF(2+K,2)=SF(2,2+K)*S(2)/S(2+K)
  SF(2+K,1)=SF(1,2+K)*S(1)/S(2+K)
  DO 8 K=1,2
    N=(K-1)*2
    SUM=0.0
    DO 6 J=1,2
      NJ=N+J
      NJ6=NJ+6
      SF(NJ6,K+4)=PF(A(NJ),B(NJ),C(NJ),D(NJ),0.0,X(K),0.0,H,X(K+1))
      SUM=SUM+S(NJ6)*SF(NJ6,K+4)
      DO 6 I=1,2
        NI=N+I+4
        NI6=NI+6
        SF(NJ6,NI6)=PF(A(NJ),B(NJ),C(NJ),D(NJ),X(K)-B(NI),X(K)-A(NI),C(NT)
1, D(NI),X(K+1))
6 SF(NI6,NJ6)=SF(NJ6,NI6)*S(NJ6)/S(NI6)
  DO 7 I=1,2
    NI=N+I+4
    NI6=NI+6
7 SF(NI6,K+2)=PF(A(NI),B(NI),C(NI),D(NI),0.0,X(K),0.0,H,X(K+1))
  SF(N+7,K+4)=SF(N+7,K+4)-SF(N+7,N+11)-SF(N+7,N+12)
  SF(N+8,K+4)=SF(N+8,K+4)-SF(N+8,N+11)-SF(N+8,N+12)
  SF(N+11,K+2)=SF(N+11,K+2)-SF(N+11,N+7)-SF(N+11,N+8)
  SF(N+12,K+2)=SF(N+12,K+2)-SF(N+12,N+7)-SF(N+12,N+8)
  SF(K+4,N+7)=SF(N+7,K+4)*S(N+7)/S(K+4)
  SF(K+4,N+8)=SF(N+8,K+4)*S(N+8)/S(K+4)
  SF(K+2,N+11)=SF(N+11,K+2)*S(N+11)/S(K+2)
  SF(K+2,N+12)=SF(N+12,K+2)*S(N+12)/S(K+2)
  SF(K+2,K+4)=(X(K)*H*SF(K+2,K+4)-SUM)/S(K+2)
  SF(K+2,K+4)=SF(K+2,K+4)-SF(K+2,N+11)-SF(K+2,N+12)
8 SF(K+4,K+2)=SF(K+2,K+4)*S(K+2)/S(K+4)
  DO 9 K=1,4
    IT=K+3
    IF(K.EQ.4)IT=3
    IW=(K-1)*2
    SUM=0.0
    DO 10 I=1,2
      IK=IW+1
      IK6=IK+6
      SF(IK6,IT)=AF(C(IK),D(IK),X(K)-B(IK),X(K)-A(IK),0.0,H,0.0,X(K+1))
      SUM=SUM+S(IK6)*SF(IK6,IT)
    DO 10 J=1,2
      IM=J+IW+2
      IF(K.EQ.4)IM=J
      IM6=IM+6
      SF(IK6,IM6)=AF(C(IK),D(IK),X(K)-B(IK),X(K)-A(IK),C(IM),D(IM),A(IM)
1, B(IM))
10 SF(IM6,IK6)=SF(IK6,IM6)*S(IK6)/S(IM6)
  DO 11 I=1,2
    IL=IW+2+I
    IF(K.EQ.4)IL=I
    IL6=IL+6
11 SF(IL6,K+2)=AF(C(IL),D(IL),A(IL),B(IL),0.0,H,0.0,X(K))
  IW7=IW+7
  IAC=IW7+2
  IWL=K+3
  IF(K.NE.4)GO TO 12
  IWL=3
  IAC=7

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```

IK=K+2
SF(IW7,IWL)=SF(IW7,IK)-SF(IW7,IAC)-SF(IW7,IAC+1)
SF(IW7+1,IWL)=SF(IW7+1,IWL)-SF(IW7+1,IAC)-SF(IW7+1,IAC+1)
SF(IAC,IK)=SF(IAC,IK)-SF(IAC,IW7)-SF(IAC,IW7+1)
SF(IAC+1,IK)=SF(IAC+1,IK)-SF(IAC+1,IW7)-SF(IAC+1,IW7+1)
SF(IWL,IW7)=SF(IW7,IWL)*S(IW7)/S(IWL)
SF(IWL,IW7+1)=SF(IW7+1,IWL)*S(IW7+1)/S(IWL)
SF(IK,IAC)=SF(IAC,IK)*S(IAC)/S(IK)
SF(IK,IAC+1)=SF(IAC+1,IK)*S(IAC+1)/S(IK)
KAC=IK+1
IF(K.EQ.4)KAC=3
SF(IK,KAC)=(X(K)*H*SF(IK,KAC)-SUM)/S(IK)
SF(IK,KAC)=SF(IK,KAC)-SF(IK,IAC)-SF(IK,IAC+1)
SF(KAC,IK)=SF(IK,KAC)*S(IK)/S(KAC)
NP=0
DO 52 I=1,14
  IB=SF(I,1)*1000.
  IF(I.EQ.1)IB=1
  IF(IB.EQ.0)GO TO 52
  NP=NP+1
  IFIL(NP)=I
  CONTINUE
DO 16 I=1,NP
DO 16 J=1,NP
  NI=IFIL(I)
  NJ=IFIL(J)
  VF(I,J)=SF(NI,NJ)
WRITE(6,307)
FORMAT(1H /4X,44H ARRAY FOR SHAPE FACTORS FROM SURFACE M TO N/1H )
WRITE(6,17)(I,I=1,NP)
FORMAT(4H M/N,15,1317)
DO 18 I=1,NP
WRITE(6,19)(I,(VF(I,J),J=1,NP))
FORMAT(1X,I2,1X,14F7.5)
STOP
END

```

```

FUNCTION PF(A1,B1,C1,D1,A2,B2,C2,D2,G)
THIS FUNCTION SUBPROGRAM CALCULATES THE SHAPE FACTOR FROM A
PLANE RECTANGULAR SURFACE TO ANOTHER PARALLEL PLANE
RECTANGULAR SURFACE.
S=(B1-A1)*(D1-C1)
IP=S*10.
IF(IP.NE.0)GO TO 1
PF=0.0
RETURN
F=F1(B1,D1,B2,D2,G)-F1(B1,D1,B2,C2,G)
1-F1(B1,D1,A2,D2,G)+F1(B1,D1,A2,C2,G)
2-F1(B1,C1,B2,D2,G)+F1(B1,C1,B2,C2,G)
3+F1(B1,C1,A2,D2,G)-F1(B1,C1,A2,C2,G)
4-F1(A1,D1,B2,D2,G)+F1(A1,D1,B2,C2,G)
5+F1(A1,D1,A2,D2,G)-F1(A1,D1,A2,C2,G)
6+F1(A1,C1,B2,D2,G)-F1(A1,C1,B2,C2,G)
7-F1(A1,C1,A2,D2,G)+F1(A1,C1,A2,C2,G)
PF=F/(3.1415927*S)
RETURN
END

```

```

FUNCTION F1(X1,Y1,X2,Y2,G)
U=X2-X1
V=Y2-Y1
UU=SQRT(G*G+U*U)
VV=SQRT(G*G+V*V)
WW=(G*G+U*U+V*V)/(G*G+V*V)
FI=U*VV*ATAN(U/VV)+V*UU*ATAN(V/UU)-G*G*LOG(WW)/2.
F1=FI/2.
RETURN
END

```

```

C
C
C
FUNCTION AF(A1,B1,C1,D1,A2,B2,C2,D2)
THIS FUNCTION SUBPROGRAM CALCULATES THE SHAPE FACTOR FROM A
PLANE RECTANGULAR SURFACE OF A ROOM TO ANOTHER PERPENDICULAR
PLANE RECTANGULAR SURFACE.
S=(B1-A1)*(D1-C1)
IP=S*10.
IF(IP,NE.0) GO TO 1
AF=0.0
RETURN
1
F=F2(A1,C1,A2,C2)+F2(B1,D1,B2,D2)
1=F2(A1,D1,B2,D2)-F2(B1,D1,B2,C2)
2+F2(A1,D1,B2,C2)-F2(B1,D1,A2,D2)
3+F2(A1,D1,A2,D2)+F2(B1,D1,A2,C2)
4-F2(A1,D1,A2,C2)-F2(B1,C1,B2,D2)
5+F2(A1,C1,B2,D2)+F2(B1,C1,B2,C2)
6-F2(A1,C1,B2,C2)+F2(B1,C1,A2,D2)
7-F2(A1,C1,A2,D2)-F2(B1,C1,A2,C2)
AF=F/(3.1415927*S)
RETURN
END

```

```

FUNCTION F2(X1,Y1,X2,Y2)
U=X2-X1
V=Y1**2+Y2**2
IF(ABS(U**2+V).LE.1.0E-7) GO TO 1
GO TO 2
1
F2=0.0
GO TO 3
2
IF(V.LE.1.0E-7) GO TO 4
GO TO 5
4
F2=.5*U**2*LOG(U**2)
GO TO 3
5
F2=.5*(U**2-V)*LOG(U**2+V)+2*U*SQRT(V)*ATAN(U/SQRT(V))
3
F2=F2/4.
RETURN
END

```

LEGEND FOR ROOM ARRANGEMENT

ASSUME WALL NO. 1 IS ON THE ROOM LENGTH

M OR N								
1	CEILING	16.00	BY	12.00				
2	FLOOR	16.00	BY	12.00				
3	WALL NO.1	16.00	BY	8.00				
4	WALL NO.2	12.00	BY	8.00				
5	WALL NO.3	16.00	BY	8.00				
6	WALL NO.4	12.00	BY	8.00				
	WINDOWS OR DOORS				A	B	C	D
7	WINDOW ON WALL 2			4.00		2.67	6.67	2.67
8	WINDOW ON WALL 3			4.00		2.67	6.67	2.67

- A - WINDOW HEIGHT
- B - WINDOW WIDTH
- C - DISTANCE OF LEFT EDGE OF WINDOW TO LEFT WALL
- D - HEIGHT FROM FLOOR TO LOWER EDGE OF WINDOW

ARRAY FOR SHAPE FACTORS FROM SURFACE M TO N

M/N	1	2	3	4	5	6	7	8
1	.00000	.36405	.18326	.11766	.16504	.13472	.01706	.01822
2	.36405	.00000	.18326	.12078	.16824	.13472	.01394	.01501
3	.27488	.27488	.00000	.12759	.15887	.13715	.00956	.01706
4	.26473	.27176	.19139	.00000	.16715	.09464	.00000	.01033
5	.27006	.27531	.17332	.12157	.00000	.14066	.01909	.00000
6	.26944	.26944	.18286	.08412	.17192	.00000	.01127	.01095
7	.30709	.25086	.11468	.00000	.21001	.10144	.00000	.01592
8	.32793	.27026	.20474	.08261	.00000	.09453	.01592	.00000

An Algorithm for Calculating Radiation Shape Factors Between Attic Surfaces Where the Attic Has a Gabled Room

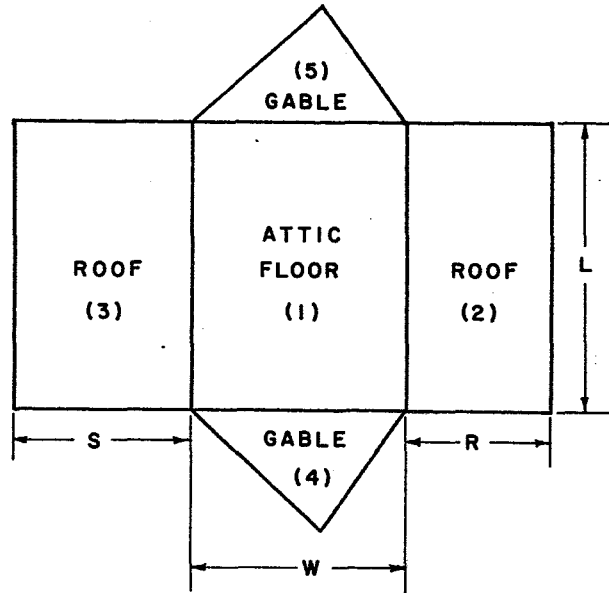


Figure A-13 Definitions of Attic Enclosure

Data:

L = Length of attic floor

W = Width of attic

* This section was contributed by B. A. Peavy and D. M. Burch, Thermal Engineering Systems Section, Center for Building Technology, National Bureau of Standards, Washington, D. C. 20234.

The primary variables determined by this subroutine are:

F_{m-n} = An array giving radiation shape factors between the various inside surfaces of an attic with a gabled roof

m, n = 1 attic floor (1)

2 roof area (2)

3 roof area (3)

4 front gable (4)

5 rear gable (5)

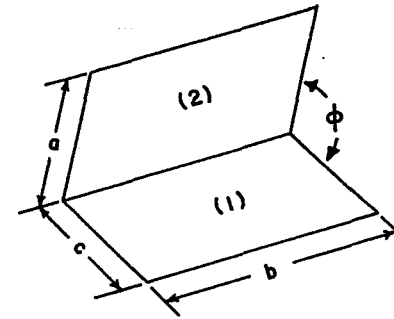


Figure A-14 Radiation Heat Exchange Between Two Adjacent Surfaces

$$X = \frac{a}{b}, Y = \frac{c}{b}, Z^2 = X^2 + Y^2 - 2XY \cos \phi$$

$$\begin{aligned}
\pi Y F_{1-2} = & -\frac{\sin 2\phi}{4} \left\{ XY \sin\phi + \left(\frac{\pi}{2} - \phi\right) (X^2 + Y^2) \right. \\
& + Y^2 \tan^{-1} \left(\frac{X - Y \cos\phi}{Y \sin\phi}\right) + X^2 \tan^{-1} \left(\frac{Y - X \cos\phi}{X \sin\phi}\right) \left. \right\} \\
& + \frac{\sin^2\phi}{4} \left\{ \left(\frac{2}{\sin^2\phi} - 1\right) \ln \left[\frac{(1+X^2)(1+Y^2)}{1+Z^2}\right] \right. \\
& + Y^2 \ln \left[\frac{Y^2(1+Z^2)}{(1+Y^2)Z^2}\right] + X^2 \cos 2\phi \ln \left[\frac{1+X^2}{1+Z^2}\right] + 2X^2 \ln \frac{X}{Z} \left. \right\} \\
& + Y \tan^{-1} \left(\frac{1}{Y}\right) + X \tan^{-1} \left(\frac{1}{X}\right) - Z \tan^{-1} \left(\frac{1}{Z}\right) \\
& + \frac{\sin\phi \sin 2\phi}{2} X \sqrt{1+X^2 \sin^2\phi} \left\{ \tan^{-1} \left[\frac{Y(1+X^2 \sin^2\phi)}{1+X^2 - XY \cos\phi}\right] \right. \\
& \left. + \cos\phi \int_0^Y \sqrt{1+\lambda^2 \sin^2\phi} \left\{ \tan^{-1} \left[\frac{X(1+\lambda^2 \sin^2\phi)}{1+\lambda^2 - \lambda X \cos\phi}\right] \right\} d\lambda \right.
\end{aligned}$$

(1)

$$\sum_{m=1}^5 F_{n-m} = 1 \quad (2)$$

$$F_{n-m} = \frac{A_m F_{m-n}}{A_n} \quad (3)$$

A_m is area of surface m

Calculation Sequence:

$$1. X = R/L, Y = W/L, \phi_1 = \cos^{-1} \left(\frac{W^2 + R^2 - S^2}{2WR} \right)$$

Compute F_{1-2}, F_{2-1} [Equations (1) and (3)]

$$2. \text{ If } S = R, \phi_2 = \phi_1, F_{1-3} = F_{1-2}, F_{3-1} = F_{2-1}, \text{ skip stage 3}$$

$$3. X = S/L, Y = W/L, \phi_2 = \cos^{-1} \left(\frac{W^2 + S^2 - R^2}{2WS} \right)$$

Compute F_{1-3}, F_{3-1} [Equations (1) and (3)]

$$4. X = S/L, Y = R/L, \phi_3 = \pi - \phi_1 - \phi_2$$

Compute F_{2-3}, F_{3-2} [Equations (1) and (3)]

$$5. F_{m-4}, F_{m-5}, m = 1, 2, 3 \text{ [Equation (2)]}$$

$$6. F_{4-m}, F_{5-m}, m = 1, 2, 3 \text{ [Equation (3)]}$$

$$7. F_{4-5}, F_{5-4} \text{ [Equation (2)]}$$

FI

An Algorithm for Approximating Inside Surface Heat Transfer Coefficients Tabulated in Table 1, Page 357 of the 1972 ASHRAE Handbook of Fundamentals

Data:

IDIR: Heat flow direction index

- 1 Upward
- 2 45° upward
- IDIR = 3 Horizontal
- 4 45° downward
- 5 Downward

e: Emittance of the surface

IV: Moving air index (IV = 0 corresponds to still air)

Calculation Sequence:

1. If IV = 0, $FI = h_c + 1.02 * e$

where

.712	1
.682	2
$h_c = .542$	for IDIR = 3
.402	4
.162	5

2. If IV ≠ 0, $FI = 2.0 \text{ Btu per (hr) (sq ft) (F)}$

FO

An Algorithm for Determining Outside Surface Heat Transfer Coefficient As a Function of Air Velocity and the Type of Surface Constructions

Data:

V: Wind velocity, knots (Determined in CLIMATE)

DIR: Wind direction (Determined in CLIMATE)

IS: Outside surface index

- 1 Stucco
- 2 Brick and rough plaster
- IS = 3 Concrete
- 4 Clear pine
- 5 Smooth plaster
- 6 Glass, white paint on pine

WA: Wall azimuth angle, degree

Calculation Sequence:

1. Conversion of the unit of wind velocity from knots into mph

$v' = 1.153 * V$

2. Outside surface heat transfer coefficient

$FO = A * (v'^{**2}) + B * v' + C$

where A, B and C are given in Table A-8

Table A-8

Value of Coefficients For Calculation
of Outside Heat Transfer Coefficient

IS	A	B	C
1	0.0	0.464	2.04
2	0.001	0.320	2.20
3	0.0	0.330	1.90
4	-0.002	0.315	1.45
5	0.0	0.244	1.80
6	-0.00125	0.262	1.45

3. Relative wind direction to the wall surface

$$RWD = WA + 180 - DIR$$

$$\text{If } |RWD| > 180, RWD = 360 - RWD$$

4. Conversion of the unit of wind velocity into m per sec

$$VV = 0.51479 * V$$

5. Air velocity close to wall surface

$$\text{If } |RWD| < 90 \text{ (windward)}$$

$$VC = 0.25 * VV \text{ for } VV > 2$$

$$VC = 0.5 \text{ for } VV \leq 2$$

$$\text{If } |RWD| \geq 90 \text{ (leeward)}$$

$$VC = 0.3 + 0.05 * VV$$

6. Convection component of the outside surface heat transfer
coefficient

$$FOC = 3.28 * ((VC) ** 0.605) *$$

This equation was derived by K. Kimura based upon the recent data published in Reference 6.

ACR*

An Algorithm for Determining Thermal Resistance
Across the Air Cavity in Walls and Roofs

Data:

DT: Temperature difference across the air space, F

L: Thickness of the air space, in.

IDIR: Heat flow direction index

1 upward

2 45° upward

IDIR = 3 horizontal

4 45° downward

5 downward

ϵ_1, ϵ_2 : Emittance of the surfaces facing the air cavity

ATC: Average temperature of the air cavity, F

Calculation Sequence:

1. Let $x = \text{Log} (DT*(L**3))$

Then using the values for $A_0, A_1, A_2, A_3,$ and A_4 which are given in Table A-9, calculate

$$y = A_0 + A_1 * x + A_2 * (x ** 2) + A_3 * (x ** 3) + A_4 * (x ** 4)*/$$

* This polynomial has been derived to represent experimental data presented in Figure 6 of Reference 7 and shown here in Figure A-15.

Table A-9 VALUES OF $A_0, A_1, A_2, A_3,$ and A_4 FOR CALCULATION OF RESISTANCE ACROSS THE AIR SPACE

IR	Range of $DT*(L**3)$	A_0	A_1	A_2	A_3	A_4
11	All the range	-1.5904	0.2824	0.0	0.0	0.0
22	$1 < DT*(L**3) \leq 10$	-1.7125	-0.0875	0.2437	-0.0420	0.0
	$10 < DT*(L**3) \leq 100$	-1.8546	0.3124	0.0	0.0	0.0
	$100 < DT*(L**3)$	-1.7380	0.2910	0.0	0.0	0.0
3	$1 < DT*(L**3) \leq 10$	-1.7410	-0.0331	0.0198	-0.0146	0.0
	$10 < DT*(L**3) \leq 100$	1.0460	-3.4660	1.5482	-0.2669	0.01673
	$100 < DT*(L**3)$	-0.2141	-0.6577	0.1693	-0.0095	0.0
4	$1 < DT*(L**3) \leq 10$	-1.7420	0.0163	-0.0409	0.0204	0.0
	$10 < DT*(L**3) \leq 100$	-6.5410	5.5710	-2.3690	0.4467	-0.0300
	$100 < DT*(L**3)$	-0.1914	6.1610	-1.3390	0.1339	-0.0050
5	$1 < DT*(L**3) \leq 10$	-1.770	0.0	0.0	0.0	0.0
	$10 < DT*(L**3)$	-1.745	0.0028	0.0029	0.0008	0.0

2. Let $z = \text{Exp} (y)$

If $z > 0.3$ $h_c = z * (1 - 0.001 * (DT-50))/L$

If $0.2 \leq z \leq 0.3$ $h_c = z * (1 + 0.00035 * (DT-50))/L$

If $z < 0.2$ $h_c = z * (1 + 0.0017 * (DT-50))/L$

3. $h_r = 0.00686 * (((ATC + 460)/100) ** 3)*$

4. $RES = 1/(h_c + (1/(1/\epsilon_1 + 1/\epsilon_2 - 1)) * h_r)$

* This polynomial has been derived to represent the curve presented in Figure 5 of Reference 7 and shown here in Figure A-16.

A Description of the Calculation Procedure for Transient Heat Conduction Using Conduction Transfer Functions

Conduction transfer functions are used widely and considered as a convenient and effective tool for the evaluation of transient heat transfer in building construction components. The conventional steady state heat transfer equation for calculating heat loss:

$$Q = U * (T_a - T_o) \tag{1}$$

where

U: overall heat transfer coefficient of roof or wall

T_a: inside air temperature

T_o: outside air temperature

is not sufficient for evaluating transient heat transfer. This equation becomes invalid because the outdoor temperature T_o usually varies as affected by solar radiation, cloud cover and wind effect. The effect of the rapid change of the outdoor temperature will not be accounted for unless the structure is extremely lightweight, such as galvanized steel.

Approximate calculation for a more accurate determination of the instantaneous heat transfer can be made by replacing T_a - T_o of equation (1) by a Total Equivalent Temperature Difference (TETD) which is usually precalculated for typical building construction components and takes into account the thermal storage effect. Although very useful, the TETD concept is only valid when the outside temperature T_o undergoes steady periodic changes. The TETD concept is therefore especially useful computing design heat transfer rates for the building where very warm or very cold

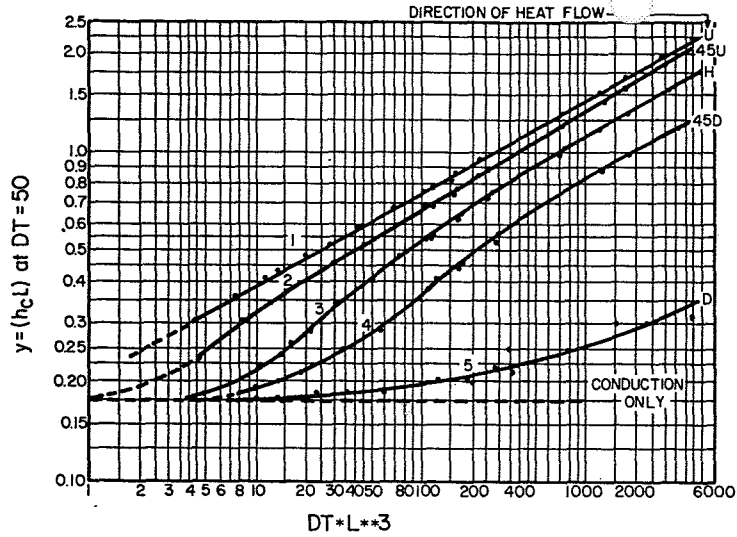


Figure A-15 Convection-Conduction Coefficient for Heat Transfer Across an Air Space for Five Orientations of the Air Space and Directions of Heat Flow

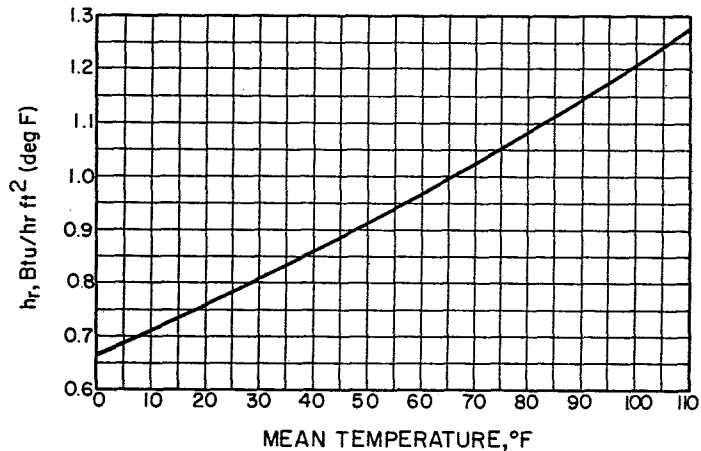


Figure A-16 Linear Radiation Coefficient for Heat Transfer Across an Air Space for a Radiation Interchange Factor

conditions are assumed to occur for several successive hrs.

A more accurate formulation for conduction heat transfer from a room to randomly fluctuating outdoor conditions is to use the hourly history of temperatures in conjunction with Conduction Transfer Functions (CTF). For example, in calculating the energy transfer from a room at an inside surface, the equation would be:

$$Q_{i,t} = \sum_{j=0}^N X_j * T_{a,t-j} - \sum_{j=0}^N Y_j * T_{o,t-j} + R * Q_{t-1} \quad (2)$$

where X_j , Y_j (for $j = 0, 1, 2 \dots N$), and R are the Conduction Transfer Functions. In equation (2), $T_{o,t-j}$ and $T_{a,t-j}$ represent outdoor and room air temperatures respectively at j th hour prior to the time for which the value of Q_t is needed. And Q_{t-1} is the heat loss Q_t , from the room at the previous hour. By having a record of $T_{o,t-1}$, $T_{o,t-2}$, $T_{o,t-3} \dots T_{o,t-j}$; and $T_{a,t-1}$, $T_{a,t-2} \dots T_{a,t-j}$; it is possible to determine the instantaneous conduction transfer, provided that the values for X_j and Y_j , and R are available.

The number of terms involved for the calculation of Q_t (or the value of N in equation (2)) depends upon the type of roof or wall construction. Generally heavy constructions require a large value, although for most conventional constructions, it seldom exceeds 20. Stephenson and Mitalas⁸ have shown that the value of N can be further decreased by employing more than one past record of Q_t (or Q_{t-j} with j being more than 1) in the following manner:

$$Q_{i,t} = \sum_{j=0}^{N'} A_j * T_{a,t-j} - \sum_{j=0}^{N'} B_j * T_{o,t-j} - \sum_{j=1}^N D_j * Q_{i,t-j} \quad (3)$$

where A_j , B_j , and D_j are the Modified Conduction Transfer Functions according to Stephenson and Mitalas⁸. Table A-10 gives values of Conduction Transfer Functions calculated for a brick wall having an overall heat transfer coefficient of 0.418. In this table, the factors designated by Z_j and C_j are the additional transfer function to be used for evaluating the instantaneous heat loss $Q_{o,t}$ at the exterior side of the structure. The applicable equations for this side of the structure are then:

$$Q_{o,t} = \sum_{j=0}^N Y_j * T_{a,t-j} - \sum_{j=0}^N Z_j * T_{o,t-j} + R * Q_{o,t-1} \quad (4)$$

or

$$Q_{o,t} = \sum_{j=0}^{N'} B_j * T_{a,t-j} - \sum_{j=0}^{N'} C_j * T_{o,t-j} - \sum_{j=1}^{N'} D_j * Q_{o,t-j} \quad (5)$$

The table also shows that for the calculation of $Q_{o,t}$ the use of Stephenson type transfer functions would permit the reduction of N from 10 to $N' = 4$ with a corresponding increase of three terms in the past record of $Q_{o,t}$.

Table A-10

Construction Data	Thermal				Thermal Resistance sq ft hr F/Btu
	Thickness (f)	Conductivity Btuh/ft	Density lb/ft ³	Specific Heat Btu/lb °F	
Inside Surface	0.000	--	--	--	.830
4-in com- mon brick	0.333	.420	100.42	.220	.793
4-in face brick	0.333	.770	125.00	.220	0.432
Outside surface	0.000	--	--	--	.330
					2.385
Time increment: 1 hour					
Overall heat transfer coefficient U = 0.418					

Conduction Transfer Functions (CTF)

j	X _j	Y _j	Z _j	R
0	.9194	.0001	1.9833	0.8398
1	-.9391	.0080	-2.1785	
2	.0606	.0243	.1983	
3	.0153	.0186	.0387	
4	.0061	.0090	.0144	
5	.0026	.0039	.0060	
6	.0011	.0017	.0026	
7	.0005	.0007	.0011	
8	.0002	.0003	.0005	
9	.0001	.0001	.0002	
10	.0000	.0001	.0001	

Stephenson Type Transfer Functions

j	A _j	B _j	C _j	D _j
0	0.9194	0.0001	1.9833	1.000
1	-1.4128	0.0079	-3.2002	-1.3552
2	0.5785	0.0202	1.3942	0.4699
3	-0.0511	0.0064	-0.1448	0.0315
4	0.0007	0.0002	0.0024	0.0002

ata:

NL: Number of layers to be considered for the analysis of a given structure: the number of layers should include surface resistance or air cavity resistance if they contribute significantly to the overall heat transfer of that particular structure

K_i: Thermal conductivity of i-th layer in Btu per (hr) (ft) (F). This value is not needed for air cavities or for the surface resistance layers.

ρ_i: Density of i-th layer in lb per (cu. ft). This value is not needed for air cavities or for the surface resistance layers.

C_i: Specific heat of i-th layer material in Btu per (lb) (F). This value is not needed for air cavities or for the surface resistance layers.

L_i: Thickness of the i-th layer in ft. This value is not needed for the air cavities or for the surface resistance layers.

RES_i: Thermal resistance of air cavities and surface resistance layers in (hr) (sq ft) (F) per Btu. This value is not needed whenever all of the remaining values such as K_i, ρ_i, C_i and L_i are given.

DT: Time increment for the conduction transfer functions
in hr (usually one hour for the building heat transfer calculations)

Subscript i refers to i-th layer and it varies from 1 to NL.

The sequence of inputting the above property values for each layer is very important and must be consistent with the particular convention adopted for the specific calculation routine. The sequence must follow in order from the inside layer to the outside layer or vice versa. It should be noted that the inclusion of the surface thermal resistance as independent layers is optional depending upon the end use of the conduction transfer functions. If the inside surface temperature is to be computed as a balance of all the heat flow involved at that surface, the thermal resistance of the inside surface should not be included in the calculation of the conduction transfer functions. The same comment applies for the outside surface.

An algorithm for the calculation of the conduction heat transfer functions will not be given here, since it involves lengthy mathematical solutions to the standard transient heat conduction differential equation. Reference (9) provides an excellent background for this calculation. Several computer programs^{10, 11/} are available for the calculation of conduction transfer functions for multi-layer walls, roofs and floor constructions. The program developed by the National Research Council of Canada^{10/} requires the layer input to be placed in order from outside toward inside. It calculates the Stephenson type conduction transfer

functions directly. The program of the National Bureau of Standards^{11/} requires the input to be placed from the inside layer first and calculates the conduction transfer functions of plane, cylindrical and spherical walls. It also calculates the transfer functions for solid objects of plane, cylindrical and spherical shapes as well as the heat conduction systems involving semi-infinite solids, approximated by basement floors and underground constructions.

Under non-steady heat conduction, the heat lost from one side of a surface is not equal to the rate of heat entry at another side. Equations (2) and (3), however, must be valid also for the steady state heat transfer problems. One of the best ways to check the consistency of the conduction transfer functions is to use them in the solution of steady state problems and see if the following criteria is met: The room side surface temperature and the outdoor side surface temperature are maintained constant for many hours so that

$$\begin{aligned} TOS_t &= TOS_{t-1} = \dots\dots TOS_{t-N} \\ TIS_t &= TIS_{t-1} = \dots\dots TIS_{t-N} \\ QI_t &= QO_t = QO_{t-1} = QI_{t-1} \end{aligned}$$

Thus

$$\begin{aligned} QO_t &= TIS_t * \sum^N Y_j - TOS_t * \sum^N Z_j + R * QO_t \\ QI_t &= TIS_t * \sum^N X_j - TOS_t * \sum^N Y_j + R * QI_t \end{aligned}$$

In order to satisfy these two equations simultaneously, it is necessary that

$$\sum_{j=0}^N X_j = \sum_{j=0}^N Y_j = \sum_{j=0}^N Z_j = U^*(1 - R)$$

HEATW

In fact, the conduction transfer functions of the sample wall shown in Table A-10 can be shown to satisfy this requirement

An Algorithm for Calculating Transient Heat Conduction Through Opaque Walls or Roofs Using Conduction Transfer Functions

$$\sum X_j = \sum Y_j = \sum Z_j = 0.0668$$

and

$$U^*(1 - R) = 0.418*(1 - 0.8398) = 0.0669$$

Data:

X_j , Y_j and Z_j for $j = 0, 1, 2, \dots, N$: Conduction transfer functions in Btu per (hr) (sq ft), (Calculated as outlined in XYZ for the system that excludes the inside and outside heat resistance layers)

R: Common ratio of the response factors X' , Y' and Z' (Calculated as outlined in XYZ),

$$R = \frac{X'_{j+1}}{X'_j} = \frac{Y'_{j+1}}{Y'_j} = \frac{Z'_{j+1}}{Z'_j} \text{ for } j \geq N$$

N: Number of the significant terms to be used for the conduction heat transfer calculation, (Calculated as outlined in XYZ)

FO_t : Outside surface heat transfer coefficient at time t, Btu per (hr) (sq ft) (F)

IT_t : Total solar radiation intensity on the outside surface at time t, Btu per (hr) (sq ft), (Calculated in SOLAD)

$$\begin{aligned} X_j &= X'_j - R * X'_{j-1} \\ Y_j &= Y'_j - R * Y'_{j-1} \\ Z_j &= Z'_j - R * Z'_{j-1} \end{aligned}$$

- TIS_{t-j}: History of inside surface temperature at times t-1, t-2, ... and (t-N)th hour, F
- TOS_{t-j}: History of outside surface temperature at times t-1, t-2, t-3, ... and (t-N)th hour, F
- DB_t: Outdoor air dry-bulb temperature at time t, F
- HEAT_{t-1}: Heat loss at the interior surface to the outdoor environment at the previous hour, Btu per (hr) (sq ft)
- QO_{t-1}: Heat loss at the exterior surface to the outdoor environment at the previous hour, Btu per (hr) (sq ft)
- a: Solar absorption coefficient at the exterior surface
- α: Cosine of the angle subtended by a vertical line and the surface normal ... (Calculated in SUN)
- TC_t: Total cloud amount ... (Calculated in CLIMATE)
- TM: A reference temperature (usually the inside design temperature), F

Calculation Sequence:

A. Exterior walls and roof

1. The heat balance equation at the exterior surface is given by

$$QR_t + QA_t + QO_t - QS_t = 0$$

where

- a) Incident solar radiation

$$QR_t = a * I T_t$$

- b) Convection heat transfer from the outdoor air

$$QA_t = FO_t * (DB_t - TOS_t)$$

- c) Conduction heat flow from the inside surface

$$QO_t = \sum_{j=0}^N Y_j * (TIS_{t-j} - TM) - \sum_{j=0}^N Z_j * (TOS_{t-j} - TM) + R * QO_{t-1}$$

- d) Heat loss to the sky

$$QS_t = 2 * \alpha * (10 - TC_t)^{**}$$

2. Let

$$SUM1 = \sum_{j=0}^N Y_j * (TIS_{t-j} - TM) + CR * QO_{t-1}$$

$$SUM2 = \sum_{j=1}^N Z_j * (TOS_{t-j} - TM)$$

3. Outside surface temperature

$$TOS_t = (QR_t - QS_t + FO_t * DB_t + SUM1 - SUM2 + Z_0 * TM) / (FO_t + Z_0)$$

4. Using this new TOS_t, the heat loss at the interior surface is then determined as follows:

$$HEAT_t = \sum_{j=0}^N X_j * (TIS_{t-j} - TM) - \sum_{j=0}^N Y_j * (TOS_{t-j} - TM) + R * HEAT_{t-1}$$

* Throughout this discussion a value of TM is always subtracted from the interior surface and exterior surface temperatures. This subtraction usually helps to minimize the digital errors which occur and are sometimes significant when a large number of numerical data are multiplied and added. Since $\sum_{j=0}^N X_j = \sum_{j=0}^N Y_j$, the net effect of the subtraction is zero.

* This expression was developed to yield a roof sky radiation of 20 Btu per (hr) (sq ft) for a cloudless condition, which was reported in Reference (12).

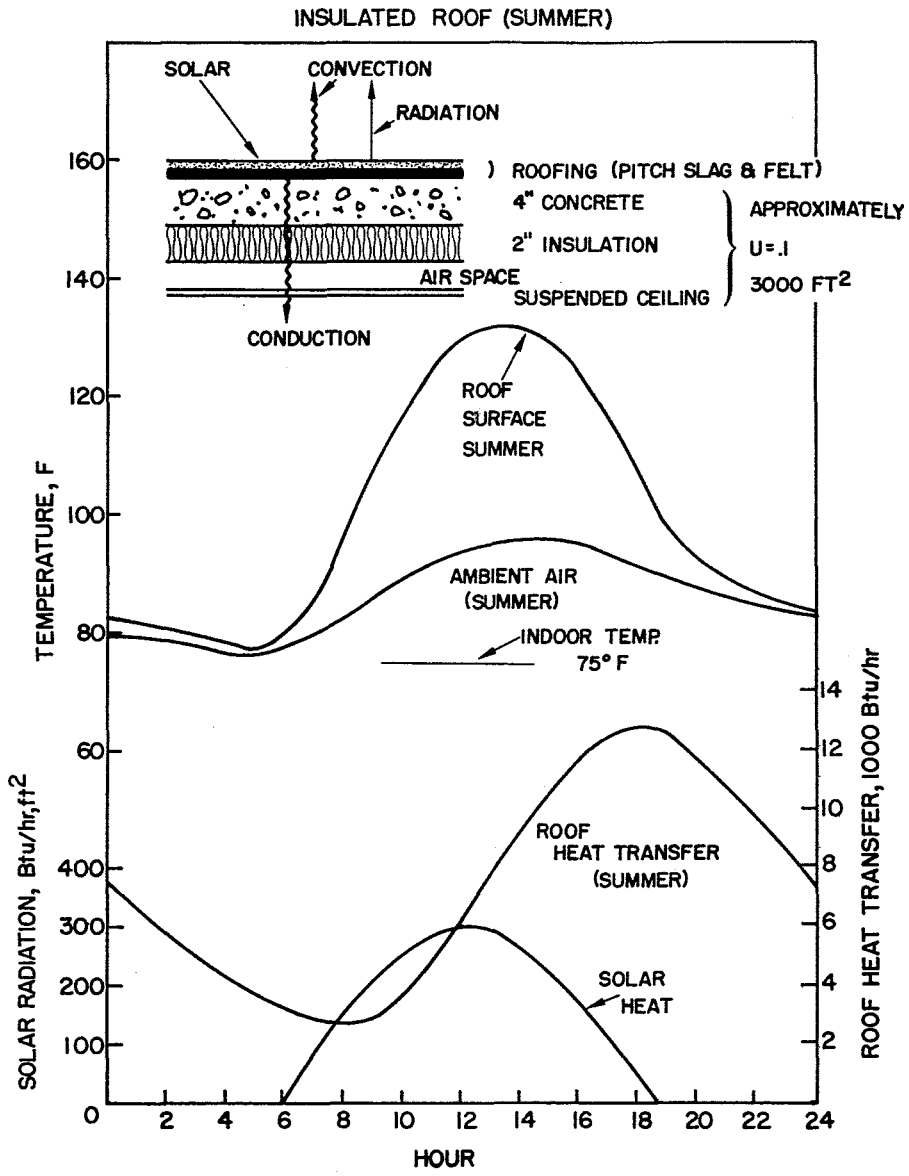


Figure A-17 Transient Heat Transfer in a Typical Roof

Table A-11

Layer No.	L(I)	K(I)	$\rho(I)$	C(I)	RES(I)	Description of Layers
1	0.	0.	0.	0.	2.04	Suspended Ceiling
2	.167	.025	13.	.320	0.	2-in. Insulation
3	.333	1.000	140.	.200	0.	4-in. Concrete
4	.031	.110	70.	.400	0.	3/4-in. Felt
5	.042	.830	55.	.400	0.	1/2-in. Pitch Slag

Time Increment $DT = 1.$

Conduction Transfer Functions

j	X_j	Y_j	Z_j	R
0	.21934	.00011	3.30513	.78793
1	-.25485	.00504	-4.27082	
2	.04650	.01060	.97885	
3	.00857	.00493	.00808	
4	.00224	.00140	.00104	
5	.00059	.00037	.00024	
6	.00016	.00010	.00006	
7	.00004	.00003	.00002	
8	.00001	.00001	.00000	
9	.00000	.00000	.00000	

Figure A-17 shows the energy balance that is involved in the above calculation sequence and the results of a typical calculation. Table A-11 gives the conduction transfer functions for the roof used in the calculations.

B. Interior walls and floor/ceiling sandwich

The calculation sequence for the partition wall and floor/ceiling sandwich is completely different from that of the exterior wall or roof. The difference is due to the fact that the air temperature at the exterior side of the construction can be assumed the same as at the interior side, at least for a climate controlled building. In order to take advantage of this fact, conduction transfer functions should be determined with the surface thermal resistance layer added at the exterior side of the structure. The heat loss through a partition wall is then calculated by

$$HEAT_t = \sum_{j=0}^N X_j * (TI_{t-j} - TM) - \sum_{j=0}^N Y_j * (TA_{t-j} - TM) + R * HEAT_{t-1}$$

where TA_{t-j} = room temperature at time (t-j)th hour

If the room temperature were maintained constant at TM , which is usually the case, the terms involving the Y_j 's would then drop out of the equation.

C. Slab on grade floor

The heat loss to the ground through the floor on grade can be calculated by using conduction transfer functions determined on the basis of flooring, concrete, and 12 inches of ground layer

$$HEAT_t = \sum_{j=0}^N X_j * (TI_{t-j} - TM) - \sum_{j=0}^N Y_j * (TG - TM) + R * HEAT_{t-1}$$

Since usually TG is constant and

$$\sum_{j=0}^N Y_j = U_G * (1 - R),$$

then

$$HEAT_t = \sum_{j=0}^N X_j * (TI_{t-j} - TM) - U_G * (1 - R) * (TG - TM) + R * HEAT_{t-1}$$

The same method is applicable to a floor with a crawl space as long as the space is not vented. The conduction transfer functions for the floor with an unvented crawl space simply has an additional air resistance layer to account for the dead air space between the floor and the ground. In many cases it is safe to assume that $TG = TM$, and then the term involving U_G would drop out of the equation.

D. Floor over the vented crawl space

The floor over a vented crawl space can be treated in the same manner as an exterior wall or roof except that the solar radiation and sky radiation terms would not be included in the energy balance and that the outside surface heat transfer coefficient is replaced by a value similar in magnitude to the inside surface heat transfer coefficient. If the conduction transfer functions include the outside surface heat transfer resistance, the calculation is simply

$$HEAT_t = \sum_{j=0}^N X_j * (TI_{t-j} - TM) - \sum_{j=0}^N Y_j * (DB_{t-j} - TM) + R * HEAT_{t-1}$$

SCHEDULE

An Algorithm to Determine Heat Gains From Lighting,
Equipment, Occupancy and Ventilation

Data:

1. Normalized 24 hour profiles ($j = 1, 2, 3 \dots 24$) of operational schedules for weekdays ($i = 1$) and weekends and holidays ($i = 2$) are given for lighting, equipment use, occupancy, ventilation, indoor temperature setting, and humidity setting as follows:

QLITE_{*i,j*}: Lighting schedules (fraction of some maximum)

QEQUP_{*i,j*}: Equipment use schedules (fraction of some maximum)

QOCUP_{*i,j*}: Occupancy schedule (fraction of some maximum)

QVENT_{*i,j*}: Ventilation fan operating schedule (fraction of some maximum)

ROOMDB_{*i,j*}: Space thermostat setting schedule

ROOMRH_{*i,j*}: Space humidistat setting schedule

2. Maximum values of the parameters to be used with the schedules

QLITX: Maximum electric power demand for lighting for the 24 hour period

QEQUPX: Maximum electric power demand for appliances for the 24 hour period, KW

QOCUPX: Maximum number of equivalent sedentary adult occupants during the 24 hour period

QVENTX: Maximum amount of ventilation air supply during the 24 hour period, cu. ft per min.

QHTWIX: Maximum amount of hot water demand during the 24 hour period, gallons per hour

3. YEAR, MONTH AND DAY

These data are needed to determine whether the day is a weekday, weekend, or holiday, and whether the day falls within daylight savings time.

4. QOS(TA): Sensible heat loss of a sedentary adult at the room temperature TA, Btu per (hr) (occupant)

QOL(TA): Latent heat loss of a sedentary adult at the room temperature TA, Btu per (hr) (occupant)

WO(DB,WB): Humidity ratio of the outdoor air for a given outdoor air dry-bulb and wet-bulb temperature, lb of water vapor per (lb of dry air)

WI(TA,WGA): Humidity ratio of the indoor air for a given dry-bulb temperature and wet-bulb temperature, lb of water vapor per (lb of dry air)

QWT: Heat needed to generate one gallon of hot water, Btu per gallon of water

Fundamentals of Room Temperature and Cooling (Heating) Load Calculations

Calculation Sequence:

1. Determine the weekday indicator IWK from WKDAY
2. Determine the holiday indicator IHOL from HOLIDAY
3. If IWK = 1 or 7 $i = 2$
If IHOL = 1 $i = 2$
Otherwise $i = 1$
4. Heat generated from lights (for $j = 1 \dots 24$), Btu per hr

$$QS_{i,j} = QLITX * 3413 * QLITE_{i,j}$$

5. Heat generated from equipment and occupants, Btu per hr (for $j = 1, 2 \dots 24$)

Sensible heat

$$QS'_{i,j} = QEQUX * 3413 * QEQUP_{i,j} + QOCUPX * QOS(TA) * QOCUP_{i,j}$$

Latent heat

$$QL_{i,j} = QOCUPX * QOL(TA) * QOCUP_{i,j}$$

6. Heat gain due to ventilation air, Btu per hr

Sensible heat

$$LEAK_{i,j} = QVENTX * C_p * 60 * (DB_j - ROOMDB_{i,j}) * QVENT_{i,j} * \rho$$

Latent heat

Determine WI_j from PSY using $ROOMDB_{i,j}$ and $ROOMRH_{i,j}$

$$LEAK'_{i,j} = QVENTX * 60 * (WO_j - WI_j) * 1060 * QVENT_{i,j} * \rho$$

C_p and ρ on this page depict specific heat and density of air in the English units.

The basic energy transfer process that occurs in a room can best be illustrated by an electrical circuit network as shown in Figure A-18. The figure represents the phenomenon in a typical room having two exterior walls, each of which contains a window, and two interior partition walls, in addition to the roof and floor (see Figure A-19). Heat conduction paths through the walls, roof, and floor are depicted by resistance and capacitance circuits and these through windows are represented by resistance circuits, implying that the windows do not have significant thermal mass. Points T_{S1} through T_{S8} in Figure A-18 indicate interior surfaces of the walls, roof, floor and windows, all of which receive conduction heat through solid material, solar radiation (represented by $\odot q$) through transparent surface and long wavelength radiation from other solid surfaces indicated by solid lines connecting the surface nodes; and they lose heat to the room air (represented by a point called TA) by the convection process (dashed lines).

At the top of Figure A-18, the radiation heat exchange between the room surfaces, the surfaces of lighting fixtures, equipment such as business machines, and occupants is depicted. Also indicated in this same location is the convective heat exchange between these items and the room air. Actual heat or power input to these internal heat sources are indicated by $\odot Q$. Although not indicated in this figure, it is possible to represent the conduction heat gain from the inner core of lighting fixtures and equipment if they have sufficient thermal mass. This equipment could of course include the unit heaters or air conditioners

* It has been assumed that there is no latent portion of the equipment heat gain. There can be exceptions to this.

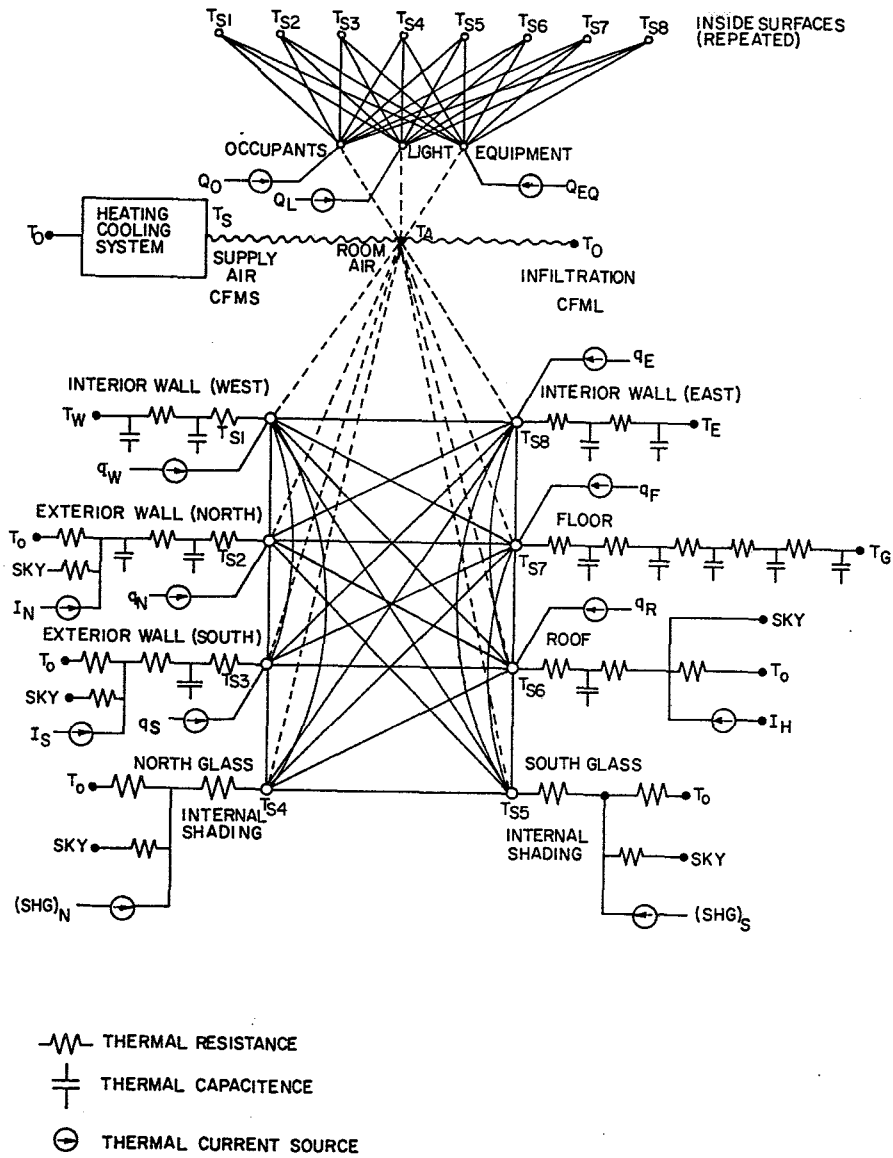


Figure A-18 Analogous Electric Circuit for Heat Exchange Process in a Room

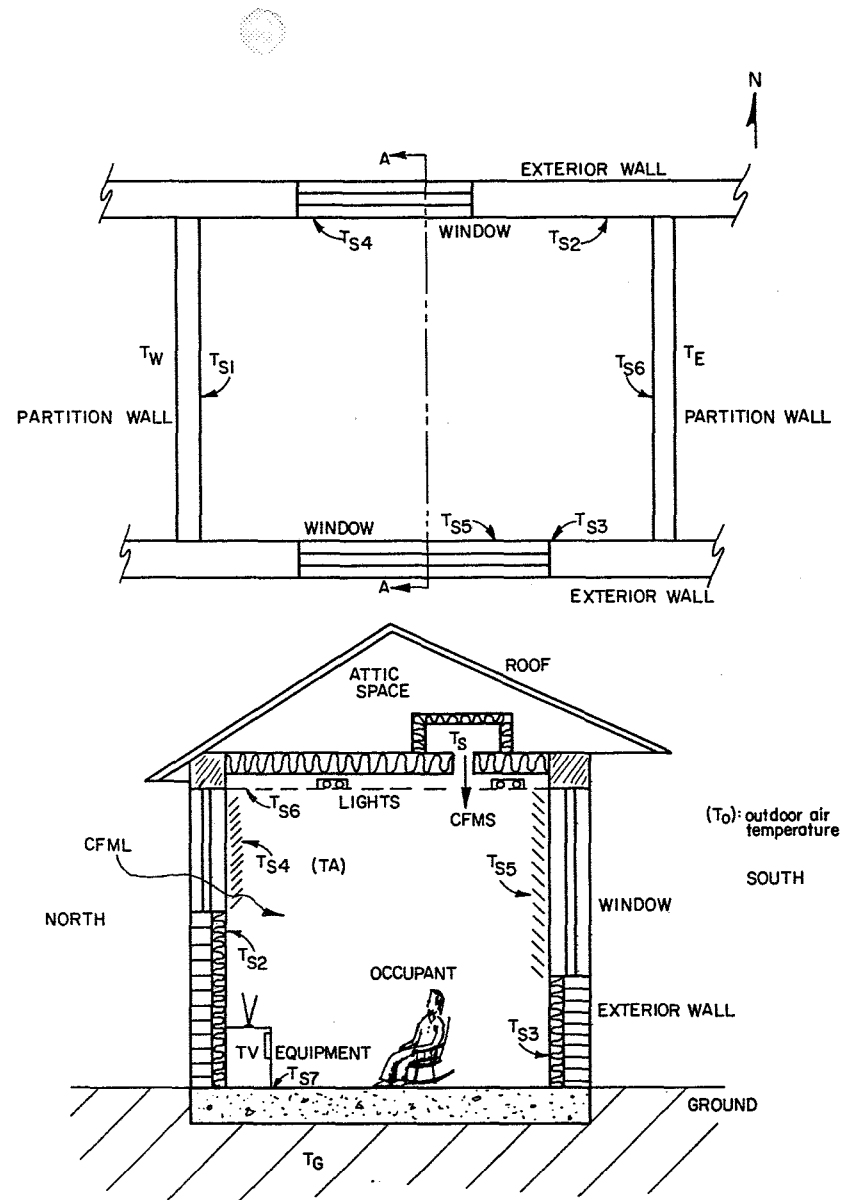


Figure A-19 Physical Model of a Typical Room Used in Figure A-18

if they are the part of the room heat exchange system. The room air changes energy with outdoor air or with the conditioned air from the central climate control system (or the forced ventilation system) and is depicted in this figure by lines $\overline{T_s T_a}$ and $\overline{T_o T_a}$. The heat exchange at the exterior surfaces with outdoor air, sky and sun (except for the partition walls and floor on grade) are also indicated using a normal design calculation procedure, the temperature or heat flow at the exterior sides of the room surfaces are usually available either by calculation or as input data. The exception occurs when two or more rooms adjacent one another are treated simultaneously. This latter case is commonly referred to as a "multi-room problem" and is very complex. No satisfactory solution for this case is presently available.

This electrical network problem can be solved and the corresponding calculation algorithm is called RMTMP (given in the following section). The solution can be obtained in two different modes: room temperature calculation mode or the room load calculation mode. The room temperature calculation mode requires the simultaneous solution of heat balance equations in order to determine all the surface temperatures together with the air temperature. On the other hand, the room cooling load calculation mode requires that the room temperature be prescribed and only the room surface temperatures are solved for. The convective heat exchange between the room air and the heat emitting surfaces is then the cooling load (or the heating load if the heat is lost to the surfaces). These two modes of computation can be combined to simulate the actual thermal behavior of a room and its environment where the temperature fluctuates. The floating temperature would be calculated

as long as it remained between prescribed limits. A heating load would be computed when the room temperature fell to the lower limit and a cooling load would be calculated when the temperature rose to the upper limit. In this manner, the calculation of load and subsequent energy requirement would more closely correspond to actual building and system operation.

RMTMP

An Algorithm to Calculate Thermal Load or Room Temperature

This routine calculates heating and cooling loads or room temperature by solving heat balance equations involving each of the room surfaces. A room surface receives conduction heat flow through the solid wall, roof or floor material from behind, convection heat flow from the air and radiation heat flow from other surfaces and internal heat source such as occupants, equipment and lighting fixtures.

Data:

NS: Total number of heat transfer surfaces contributing to the room heat balance

S_i : Area of i-th heat transfer surface, sq ft, where $i = 1, 2, 3, \dots NS$

$X_{i,j}$, $Y_{i,j}$ and $Z_{i,j}$:
Where $i = 1, 2, 3, \dots NS$
 $j = 1, 2, \dots N_i$

Conduction transfer functions of i-th surface in Btu per (hr) (sq ft) ... (Calculated in XYZ).

These conduction transfer functions are usually evaluated without the interior or room side surface thermal resistance. The thermal resistance layer of exterior surface is also omitted if the exterior surface temperature is to be computed as a result of a heat balance involving solar radia-

tion, sky radiation and convective loss to the outdoor air.

N_i : Number of conduction transfer function terms to be used for the calculation of the i-th surface conduction heat gain

R_i : Common ratio for the conduction transfer function of the i-th surface

$TOS_{i,t-j}^*$: for $i = 1, 2, 3, \dots NS$ and $j = 0, 1, 2, \dots N_i$
Outside surface temperature history from present hour to that of N_i hours ago for i-th surface, F.
(This information is available from HEATW routine.)

$TIS_{i,t-j}$: for $i = 1, 2, 3, \dots NS$ and $j = 1, 2, 3, \dots N_i$
Inside surface temperature history from one hour ago to N_i hours ago for i-th surface, F. The present value (for $j = 0$) will be computed in this routine and stored for future use.

TA_t : Air temperature of the room at time t, F

DB_t : Outdoor air temperature at time t, F

TS_t : Supply air temperature from the central system at time t, F

H_i : Inside surface convection heat transfer coefficient for i-th surface, Btu per (hr) (sq ft) (F)

* Subscript t refers to the present time t and t-j refers to the present time minus j hours.

$F_{i,k}$: Radiation heat exchange view factor between the i-th surface and k-th surface

$$F_{i,k} = F_{k,i} = 0$$

E_i : Emissivity of the i-th surface

$R_{i,t}$: Radiant heat flux impinging upon i-th surface at time t from various sources, which include solar radiation, radiation from lights, occupants and equipment, Btu per (hr) (sq ft)

$Q_{i,t}$: Heat conducted into i-th surface at time t, Btu per (hr) (sq ft)

GL_t : Mass air flow rate due to air leakage at time t, lb per hr

GS_t : Mass air flow rate of the supply air from the central system at time t, lb per hr

QEQU: Internal heat generated from equipment such as business machines and computers, Btu per hr

QOCPS: Internal heat (sensible) generated from occupants (a function of room air temperature), Btu per hr

QLITE: Heat from lights, Btu per hr

RE: Fraction of internal heat gain from equipment that can be assumed to be convective

RO: Fraction of internal heat gain from occupants that can be assumed to be convective

RL: Fraction of heat gain from lights that can be assumed to be convective

$SHG_{i,t}$: Solar incident radiation on i-th surface at time t, Btu per (hr) (sq ft)

Calculation Sequence:

1. Heat balance equation at the i-th surface at time t

$$Q_{i,t} = \sum_{j=0}^{N_i} X_{i,j} * TIS_{i,t-j} - \sum_{j=0}^{N_i} Y_{i,j} * TOS_{i,t-j} + R_{i,t} * Q_{i,t-1}$$

$$= H_{i,t} * (TA_t - TIS_{i,t}) + \sum_{k=1}^{NS} G_{i,k} * (TIS_{k,t} - TIS_{i,t}) + R_{i,t}$$

where $G_{i,k} = 4 * E_i * F_{i,k} * (TA_t + 460) ** 3 * 0.1714 * 10 ** -8$

$$R_{i,t} = SHG_{i,t} + \frac{((1-RE)*QEQU + (1-RO)*QOCPS + (1-RL)*QLITE)}{\sum_{i=1}^{N_s} S_i}$$

2. Heat balance for the room air

$$\sum_{i=1}^{N_s} S_i * (TIS_{i,t} - TA_t) + GL_t * C_p * (DB - TA_t) + GS_t * C_p * (TS_t - TA_t) + QEQU * RE + QOCPS * RO + QLITE * RL = 0$$

$$(TS_t - TA_t) + QEQU * RE + QOCPS * RO + QLITE * RL = 0$$

where C_p is the specific heat of air in Btu per (lb) (F)

The values of GS_t and TS_t , supply air flow rate and its temperature, are the link between the load calculation and the system simulation. (More detailed explanation of this aspect is given in the final portion of this section.)

3. Assigning matrix elements for $i = 1, 2, 3, \dots$ and for $k = 1, 2, 3, \dots, NS$

$$A_{i,i} = X_{i,1} + H_i + \sum_{k=1}^{NS} G_{i,k}$$

$$A_{i,k} = -G_{i,k} = A_{k,i} = -G_{k,i}$$

$$A_{i,NS+1} = -H_i$$

$$B_i = -\sum_{j=1}^{N_i} X_{i,j} * TIS_{i,t-j} + \sum_{j=0}^{N_i} Y_{i,j} * TOS_{i,t-j} - R * Q_{i,t-1} + R_i,$$

$$A_{NS+1,k} = S_k * H_k$$

$$A_{NS+1,NS+1} = -(GL_t + GS_t) * C_p - \sum_{k=1}^{NS} H_k * S_k$$

$$B_{NS+1} = -QEQU*RE + -QOCPS*RO - QLITE*RL - GL_t * C_p * DB_t - GS_t * C_p * TS_t$$

4. Using these matrix elements, the following NS+1 equation should be solved simultaneously for $TIS_{i,t}$ ($i = 1, 2, \dots, NS$) and for TA_t

$$\begin{bmatrix} A_{1,1} & A_{1,2} & A_{1,NS+1} \\ A_{2,1} & A_{2,2} & A_{2,NS+1} \\ \dots & \dots & \dots \\ A_{NS,1} & A_{NS,2} & A_{NS,NS+1} \\ A_{NS+1,1} & A_{NS+1,2} & A_{NS+1,NS+1} \end{bmatrix} * \begin{bmatrix} TIS_{1,t} \\ TIS_{2,t} \\ \dots \\ TIS_{NS,t} \\ TA_t \end{bmatrix} = \begin{bmatrix} B_1 \\ B_2 \\ \dots \\ B_{NS} \\ B_{NS+1} \end{bmatrix} *$$

5. When the value of TA_t has been specified, as in the case of a controlled condition, the following NS equations should be solved instead of the NS+1 equations given above

$$\begin{bmatrix} A_{1,1} & A_{1,2} & \dots & A_{1,NS} \\ A_{2,1} & A_{2,2} & & A_{2,NS} \\ \dots & \dots & & \dots \\ A_{NS,1} & A_{NS,2} & & A_{NS,NS} \end{bmatrix} * \begin{bmatrix} TIS_{1,t} \\ TIS_{2,t} \\ \dots \\ TIS_{NS,t} \end{bmatrix} = \begin{bmatrix} B'_1 \\ B'_2 \\ \dots \\ B'_{NS} \end{bmatrix} *$$

where

$$B'_i = B_i - A_{i,NS+1} * TA_t$$

* Remark: The matrix type solution is not required if it is assumed that

$$\sum_{k=1}^{NS} G_{i,k} * (TIS_{k,t} - TIS_{i,t}) = HR * (TA_t - TIS_{i,t})$$

or

$$A_{i,k} = 0 \text{ when } i \neq k$$

6. Calculate the sensible load by

$$QLS_t = \sum_{i=1}^N S_i * (TIS_{i,t} - TA_t) + GL_t * C_p * (DB_t - TA_t) \\ + QEQUP * RE + QOCPS * RO + QLITE * RL$$

In this expression, QL is a cooling load if positive and it is a heating load if negative. This is the heat picked up by the room air (or that lost by the room air) which has to be removed (or added) by the central air conditioning system.

Note that for ordinary load calculations, GS_t and TS_t are not used as long as the following condition is satisfied:

$$|QLS_t| \leq |GS_t * C_p * (TA_t - TS_t)| \dots \text{Maximum capacity} \\ \text{of the heating or cooling system}$$

In other words, the desired or prescribed room temperature can be maintained as long as the calculated load is less than the maximum capacity of the central system. When the above condition is not satisfied because of the inadequate values for either the air supply rate or the supply air temperature, the room temperature used for the load calculation is no longer valid. The calculation must then be revised, first calculating the room temperature as outlined in 3 above.

7. Latent Load

If moisture condensation and absorption by room walls, or drying of the wall panels can be neglected, the latent load is the same as the latent heat gain or loss, provided the following condition is met:

$$QLL_t \leq GS_t * \lambda * (WA - WS), \text{ where } \lambda = \text{latent heat} \\ \text{of vaporization } \approx 1061 \text{ Btu per lb of water}$$

In other words, the desired moisture level can be maintained as long as the latent load of the room or the building is less than the capacity of the central system to remove (add) water vapor. When the above condition is not satisfied because of the inadequate air flow rate of the air supply system or the value of WS, the room humidity ratio would change according to

$$WI = \frac{GS_t * WS + GL_t * WA + QLL/\lambda}{GS_t + GL_t}$$

This equation becomes indeterminate when there is no air supply or air leakage to or from the room. Theoretically the room relative humidity would reach 100% soon after the air supply or air leakage is stopped provided normal internal sources were still present. Under those conditions, seasonal value of WI to be used would be that corresponding to the dew point temperature which would be approximately equal to the average

wall surface temperature of the space.

The most important application of RMTMP is for taking into account some of the performance characteristics of the room's (or space's) heating and cooling systems where the evaluation of heating and cooling load is linked to the system capacity. Presented in this section is a sample algorithm to illustrate how RMTMP can be used to account for the type of occupancy, temperature control scheme, and the system capacity. In this illustration, the heating/cooling load will be set equal to the maximum capacity of the system when the calculated load at a given time is greater than the maximum system capacity. The space or room temperature is then calculated on the basis of net load, which is the difference between the calculated load at a given design temperature and the maximum system capacity. If, on the other hand, the space temperature falls within the prescribed upper and the lower limits, the load is set equal to zero. The same procedure is applied to the latent load calculation. The details of the algorithm is depicted in the flow diagram (Figure A-20). The nomenclature for the figure is given in Table A-12.

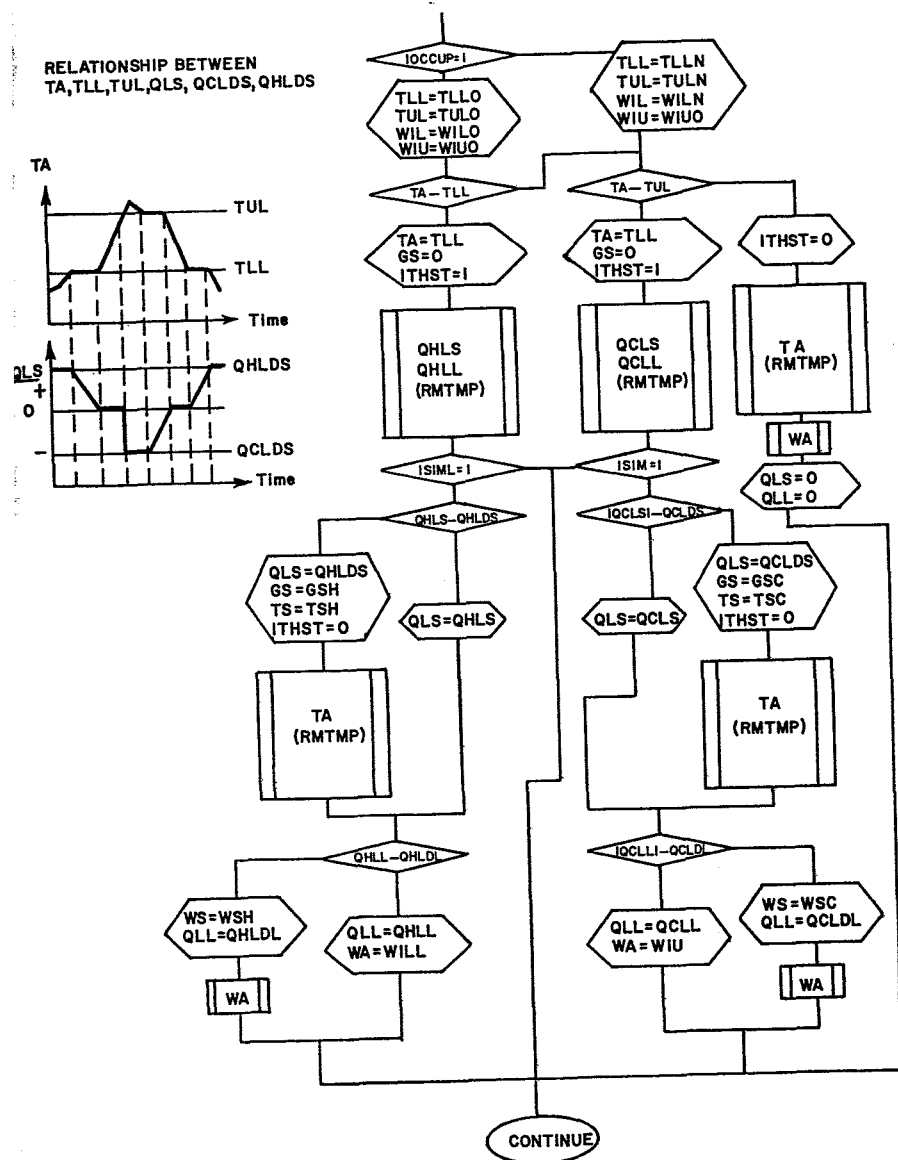
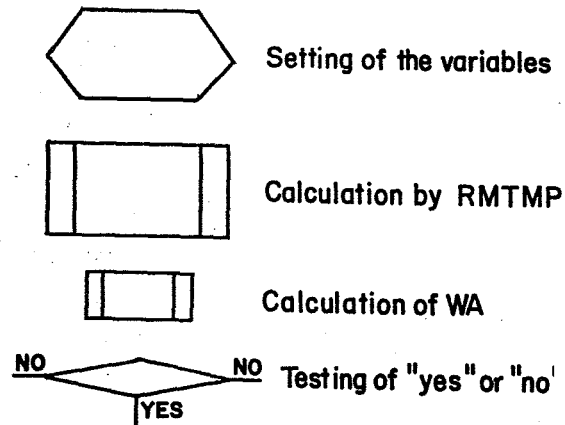


Figure A-20 Flow Diagram of the Room Temperature and the Room Thermal Load Calculation Steps



Nomenclature for Figure A-20

- IOCCUP: Occupancy index
- 1 if during the occupied period
 - 0 if during the unoccupied period
- LSIML: System capacity consideration index
- 1 if the calculated load is to be compared with the maximum capacity of the system
 - 0 if the load is to be estimated without regard to the installed system capacity
- ITHST: Room temperature control index
- 1 if a space temperature is prescribed at a given level and the heating and cooling load is to be calculated to meet this prescribed condition
 - 0 if the space temperature is to be determined as a balance between the required load and the available capacity of the system
- TLL: The lower limit of temperature, below which heating must be supplied
- TLL0: The lower limit of temperature during the occupied period
- TLLN: The lower limit of temperature during the unoccupied period
- TUL: The upper limit of temperature, above which cooling must be provided

TULO: The upper limit of temperature during the occupied period

TULN: The upper limit of temperature during the unoccupied period

WIL: The lower limit of humidity ratio, below which the room requires humidification

WILO: The lower limit of humidity ratio during the occupied period

WILN: The lower limit of humidity ratio during the unoccupied period

WIU: The upper limit of humidity ratio, above which the room requires dehumidification

WIUO: The upper limit of humidity ratio during the occupied period

WIUN: The upper limit of humidity ratio during the unoccupied period

GS: Mass flow rate of air from the supply system

GSH: Maximum mass flow rate of air from the supply system during heating

GSC: Maximum mass flow rate of air from the supply system during cooling

TS: Air temperature of the supply air system

TSH: Maximum temperature of air from the supply system during heating

TSC: Minimum temperature of air from the supply system during cooling

WS: Humidity ratio of the air from the supply system

WSH: Maximum humidity ratio of air from the supply system during a period when heating and humidification occur

WSC: Minimum humidity ratio of air from the supply system during a period when cooling and dehumidification occur

C_p : Specific heat of air

λ : Latent heat of vaporization

QHLDS: Maximum system capacity for heating, Btu per hr

$$QHLDS = GSH * C_p * (TSH - TA)$$

QHLDL: Maximum system capacity for humidification

$$QHLDL = GSH * \lambda * (WSH - WA)$$

QCLDS: Maximum system capacity for cooling, Btu per hr

$$QCLDS = GSC * C_p * (TA - TSC)$$

QCLDL: Maximum system capacity for dehumidification

$$QCLDL = GSC * \lambda * (WA - WSC)$$

The primary variables that are determined by this mode of calculation are:

QHL: Sensible heating load of the space

QHLL: Latent heating load of the space

QCLS: Sensible cooling load of the space

QCLL: Latent cooling load of the space

TA: Space temperature

WA: Space humidity ratio

ATTIC

A Description of the Load Calculation for an Attic Space

In many cases the heating and cooling load in an attic space is affected strongly by the manner in which the attic conditions are maintained. The non-ventilated attic space may be treated as a simple air space within a roof structure and accounted for in the calculation of the conduction transfer functions. Where the attic space is ventilated during the summer to take advantage of the resulting natural cooling effect of the outdoor air, it has to be treated somewhat differently. Since it is reasonable to assume that the radiation heat exchange between the underside of the roof surface and the attic is as significant as the ventilation air rate in determining the attic thermal condition, RMTMP should be used. No additional algorithms are then required since all aspects covered in RMTMP apply directly to the attic heat transfer calculation. Of course, the solar radiation through windows, and internal heat gain from lighting and occupants would most likely be omitted. The floor should be treated as having its exterior surface exposed to the environmental conditions of the room below. The heat loss at the "exterior" surface of the floor then becomes the heat gain to the room beneath the ceiling.

FIJ 1 outlines the calculation procedure for obtaining the necessary shape factors where the attic has a gabled roof.

IHG

An Algorithm to Calculate Instantaneous Heat Gain of a Space at Time t

Data:

Windows:

- NY: Number of windows
- AY_k: Area of each window, sq ft
- UY_k: Overall heat transfer coefficient for each window, Btu per (hr) (sq ft) (f)
- SHG_k: Solar heat gain through each window, Btu per (hr) (sq ft) (Calculated in SHG)

Exterior Walls and Roofs:

- NX: Number of exterior walls and roofs
- AX_k: Area of each exterior wall and roof, sq ft
- HEAT_k: Heat gain through each exterior wall and roof, Btu per (hr) (sq ft) (Calculated in HEATW)
where $k = 1, 2, \dots NX$

Lights:

- NS: Number of different types of lights
- QS_k: Power input to each type of light, Btu per hr
where $k = 1, 2, \dots NS$

Internal Heat Source Other than Lights:

- NS': Number of different types of internal sensible heat sources other than lights
- QS_k': Heat generation from each internal sensible heat source, where k = 1, 2, ... NS'
- NL: Number of different types of internal latent heat sources
- QL_k: Latent heat gain from each internal latent heat source, Btu per hr, where k = 1, 2, ... NL

Inside Doors:

- ND: Number of inside doors
- AD_k: Area of each inside door, sq ft
- UD_k: Overall heat transfer coefficient of each inside door, where k = 1, 2, ... ND'

Outside Doors:

- ND': Number of outside doors
- AD_k': Area of each outside door, sq ft
- UD_k': Overall heat transfer coefficient of each outside door, Btu per (hr) (sq ft) (F), where k = 1, 2, ... ND'

Partitions:

- NP: Number of partitions which separate the space from other spaces at different temperatures
- AP_k: Area of each of these partitions, sq ft
- UP_k: Overall heat transfer coefficient for each of these partition walls, Btu per (hr) (sq ft) (F), where k = 1, 2, ... NP

Underground Walls:

- NG: Number of underground walls
- AG_k: Area of each underground wall, sq ft
- UG_k: Overall heat transfer coefficient of each underground wall, Btu per (hr) (sq ft) (F), where k = 1, 2, ... NG

Underground Floors:

- NGF: Number of underground floors
- AGF_k: Area of each underground floor, sq ft
- UGF_k: Overall heat transfer coefficient of each underground floor, Btu per (hr) (sq ft) (F), where k = 1, 2, ... NGF

Internal Infiltration

- NLK: Number of internal air leakage sources
- LEAK_k: Air leakage from each source, cfm (Calculated in INFIL), where k = 1, 2, ... NLK

External Infiltration:

NLK': Number of external air leakage sources

LEAK'_k: Air leakage from each external source, cfm

(Calculated in INFIL), where k = 1, 2, ...

NLK'

Temperatures:

TA'_k: Dry-bulb temperature of each adjacent space, F, where k = 1; 2, ... ND, NP or NLK

DB: Outside air dry-bulb temperature, F (Obtained from CLIMATE)

TG: Average ground water temperature at half underground basement depth, F

TGW: Ground water temperature, F

TZ: Space dry-bulb temperature, F

Humidity Ratios:

WA'_k: Humidity ratio of adjacent space, lb water per lb dry air, where k = 1, 2, ... ND, NP or NLK

WO: Outside air humidity ratio, lb water per lb dry air (Calculated in PSY)

WZ: Space humidity ratio, lb water per lb dry air

The following heat gains are calculated in this subroutine:

HEATG Total hourly solar heat gain through windows, Btu or : HEATG' per hr

HEATX: Total hourly heat gain through exterior walls and roofs, Btu per hr

HEATIS: Total power input to lights, Btu per hr

HEATDP: Total sensible heat gain due to heat transfer through doors, partitions, underground walls and floors, and internal heat sources other than lights, Btu per hr

HEATVS: Total hourly sensible heat gain due to infiltration, Btu per hr

HEATL: Total hourly latent heat gain due to infiltration and internal heat sources, Btu per hr

Calculation Sequence:

1. HEATG NY or HEATG' = Σ AY'_k*SHG'_k k=1

2. HEATX = Σ AX'_k*HEAT'_k k=1 NX

3. HEATIS = Σ QS'_k k=1 NS

4. HEATDP = Σ AD'_k*UD'_k*(TA'_k-TZ) + Σ AD'_k*UK'_k(DB-TZ) k=1 ND'

A Simplified Procedure for Obtaining Approximate Cooling Load
by the Use of Weighting Factors

The procedure presented here was developed by Mitalas and Stephenson

of National Research Council of Canada^{13, 14/} in order to expedite the otherwise complex and time-consuming solution of the heat balance simultaneous equations. The rigorous solution similar to that described in RTMP was first obtained for typical rooms in commercial buildings with pulse type excitations that simulate various heat gains. The solution for these pulse excitations were then converted into new types of transfer functions called Weighting Factors. Weighting Factors developed for typical office spaces of light, medium, and heavy constructions are shown in Tables A-13, A-14, and A-15 for solar heat gain with no internal shading devices; heat gain conduction through interior and exterior structure components, solar heat gain with interior shading devices and all internal sources except lighting; and the heat gain due to lighting. By multiplying these Weighting Factors to the history of respective heat gains in a convolution scheme, similar to the way the conduction transfer functions are multiplied to the temperature history, it is possible to calculate an approximate cooling load.

Data:

$$AG_j \text{ for } j = 0, 1, 2 \dots MG \text{ and } BG_j \text{ for } j = 1, 2, 3 \dots MG'$$

Weighting Factors for the solar heat gain HEATG
(no internal shading devices)

$$\begin{aligned}
 & + \sum_{k=1}^{NY} AY_k * UY_k * (DB-TZ) + \sum_{k=1}^{NG} AG_k * UG_k * (TG-TZ)^* \\
 & + \sum_{k=1}^{NGF} AGF_k * UGF_k * (TGW-TZ) + \sum_{k=1}^{NP} AP_k * UP_k * (TA_k - TZ) \\
 & + \sum_{k=1}^{NS'} QS'_k \\
 5. \text{ HEATVS} &= 1.08 * \left(\sum_{k=1}^{NLK} LEAK_k * (TA_k - TZ) + \sum_{k=1}^{NLK'} LEAK'_k * (DB-TZ) \right)^{**} \\
 6. \text{ HEATL} &= 4775 * \left(\sum_{k=1}^{NLK} LEAK_k * (WA_k - WZ) + \sum_{k=1}^{NLK'} LEAK'_k * (WO - WZ) \right) \\
 & + \sum_{k=1}^{NL} QL_k^{**}/
 \end{aligned}$$

* The values of UG given in the 1972 ASHRAE Handbook of Fundamentals are based on TGW. A program for calculating basement wall losses using TC has been developed at the National Bureau of Standards. When using present ASHRAE values for UG, use TGW instead of TG.

** The coefficients 1.08 and 4775 in these equations are valid for the standard air density. If desired, they can be adjusted to actual conditions by multiplying both of them by $\frac{\rho}{0.075}$, where ρ is the actual density of the air expressed in lb per cu ft.

AX_j for $j = 0, 1, 2 \dots MX$ and BX_j for $j = 1, 2 \dots MX'$

Weighting factors for

HEATX: Conduction heat gain

HEATG: Solar heat gain where there are internal shading devices

HEATDP: Heat gain due to air leakage and internal sources except lighting

AIS_j for $j = 0, 1, 2 \dots MIX$ and BIS_j for $j = 1, 2, 3 \dots MIS'$

Weighting factors for the heat gain from lighting HEATIS

In order to make use of the weighting factor concept, it is necessary to have previous values of heat gains as well as values of cooling loads. By denoting the cooling load due to HEATG as HLCG, due to HEATX, HEATG', and HEATDP as HLCX and that due to HEATIS as HLCIS, the following set of the previous data are needed:

HEATG _{t-j}	for $j = 0, 1, 2 \dots MG$
HEATX _{t-j} ; HEATG' _{t-j} ; HEATDP _{t-j}	for $j = 0, 1, 2 \dots MX$
HEATIS _{t-j}	for $j = 0, 1, 2, 3 \dots MIS$
HLCG _{t-j}	for $j = 1, 2, 3 \dots MG'$
HLCX _{t-j}	for $j = 1, 2, 3 \dots MX'$
HLCIS _{t-j}	for $j = 1, 2, 3 \dots MIS'$

Calculation Sequence:

- Using the Weighting Factors* given in Tables A-13, A-14, and

* The Weighting Factors given in Tables A-13, A-14, and A-15 are for typical office construction. They are obtained using the method described in Appendix B.

A-15 and factor defined by equation "d", calculate load components corresponding to the heat gains

$$\begin{aligned}
 \text{a. } HLCG_t &= Fc \sum_{j=0}^{MG} AG_j * HEATG_{t-j} - \sum_{j=1}^{MG'} BG_j * HLCG_{t-j} \\
 \text{b. } HLCX_t &= Fc \sum_{j=0}^{MX} AX_j * (HEATX_{t-j} + HEATG'_{t-j} + HEATDP_{t-j}) \\
 &\quad - \sum_{j=1}^{MX'} BX_j * HLCX_{t-j} \\
 \text{c. } HLCIS_t &= Fc \sum_{j=0}^{MIS} AIS_j * HEATIS_{t-j} - \sum_{j=1}^{MIS'} BIS_j * HLCIS_{t-j}
 \end{aligned}$$

The coefficients given in Tables A-13, A-14, and A-15 are for the case where all the heat gain energy appears eventually as cooling load. In most cases, a fraction of the input is lost to the surroundings. This fraction depends on the thermal conductance between the room air and the surroundings. One estimate of this fraction F_c , is given by

$$\text{d. } F_c = 1 - 0.02 K_T \dots$$

for the range $1.0 > F_c > 0.7$

$$\begin{aligned}
 \text{where } K_T &= \frac{1}{L_F} (U_{\text{window}} A_{\text{window}} \\
 &\quad + U_{\text{exterior wall}} A_{\text{exterior wall}*} \\
 &\quad + U_{\text{corridor wall}} A_{\text{corridor wall}})
 \end{aligned}$$

* A $U \cdot A$ product should also be included for walls that adjoin unconditioned spaces even though the walls are not exterior ones.

L_F = Length of room exterior perimeter
 U = U value of the room enclosure element
 A = Area of the room enclosure element

2. Hourly load

a. Sensible load

$$SCL_t = HLCG_t + HLCX_t + HLCIS_t + HEATVS_t$$

b. Latent load

$$= HEATL_t$$

Table A-13

WEIGHTING FACTORS FOR HEATG

	Weighting Factor Symbol	Heavy* Structure	Medium* Structure	Light* Structure
MG = 1	AG ₀	0.187	0.197	0.224
	AG ₁	-0.097	-0.067	-0.044
MG' = 1	BG ₀	1.00	1.00	1.00
	BG ₁	-0.91	-0.87	-0.82

- * Heavy Structure - 6" concrete floor slab, 6" concrete exterior wall, approximately 130 lb of building material per sq ft of floor area.
- Medium Structure - 4" concrete floor slab, 4" concrete exterior wall, approximately 70 lb of building material per sq ft of floor area.
- Light Structure - 2" concrete floor slab, exterior frame wall approximately 30 lb of building material per sq ft of floor area.

Table A-14

NORMALIZED WEIGHTING FACTORS FOR HEATX + HEATG' + HEATDP

	Weighting Factor Symbol	Heavy Structure	Medium Structure	Light Structure
MX = 1	AX ₀	0.676	0.681	0.703
	AX ₁	-0.586	-0.551	-0.523
MX' = 1	BX ₀	1.00	1.00	1.00
	BX ₁	-0.91	-0.87	-0.82

Table A-15

WEIGHTING FACTORS FOR HEATIS

Weighting Factor Symbol	Heavy Structure	Medium Structure	Light Structure
Fluorescent fixtures recessed into a suspended ceiling, ceiling plenum not vented.			
MIS = 2			
AIS ₀	0.00	0.00	0.00
AIS ₁	0.53	0.53	0.53
AIS ₂	-0.44	-0.40	-0.35
* Fluorescent fixtures recessed into a suspended ceiling, return air through ceiling plenum.			
MIS = 2			
AIS ₀	0.00	0.00	0.00
AIS ₁	0.59	0.59	0.59
AIS ₂	-0.50	-0.46	-0.41
Fluorescent fixtures recessed into a suspended ceiling, supply and return air through fixtures.			
MIS = 2			
AIS ₀	0.00	0.00	0.00
AIS ₁	0.87	0.87	0.87
AIS ₂	-0.78	-0.74	-0.69

* Manufacturer's data sheet must be consulted to obtain the fractions of light input energy that are picked up by the room air and by ventilation air in the ceiling plenum.

Table A-15 continued

Weighting Factor Symbol	Heavy Structure	Medium Structure	Light Structure
Incandescent lights exposed in the room air.			
MIS = 2			
AIS ₀	0.00	0.00	0.00
AIS ₁	0.50	0.50	0.50
AIS ₂	-0.41	-0.37	-0.32
The "BIS" Coefficients.			
MIS' = 1			
BIS ₀	1.00	1.00	1.00
BIS ₁	-0.91	-0.87	-0.82

RMRT

An Algorithm for Calculating Weighting
Factors for Space Air Temperature

This algorithm provides a sample calculation method for obtaining the weighting factors for deviation of space temperature for the design value; the value at which the space heating/cooling loads are obtained for by the HLC routine.

This general algorithm illustrated here is for a space enclosure surrounded with spaces on both sides as well as above and below that are thermally at the same conditions. The space enclosure consists of an external wall, interior partition walls, corridor partition wall, ceiling, floor, furnishings, an outside door and a window.

Data:

AF: Floor area, sq ft
AC: Ceiling area, sq ft
AP: Interior partition wall area, sq ft
AK: Corridor wall area, sq ft
AW: Exterior wall area, sq ft
AG: Window glass area, sq ft
AD: Door area, sq ft
AFN: Internal furnishings area, sq ft
BF_j: Transfer functions for floor, Btu per (hr) (sq ft)
CF_j: (F), (Calculated in XYZ)
DF_j

BC_j: Transfer functions for ceiling, Btu per (hr) (sq ft)
CC_j: (F), (Calculated in XYZ)
DC_j
BK_j: Transfer functions for corridor wall, Btu per (hr)
DK_j: (sq ft) (F), (Calculated in XYZ)
BP_j: Transfer functions for interior partition walls, Btu
CP_j: per (hr) (sq ft) (F), (Calculated in XYZ)
DP_j
CW_j: Transfer functions for exterior walls, Btu per (hr)
DW_j: (sq ft) (F), (Calculated in XYZ)
CD_j: Transfer functions for outside door, Btu per (hr)
DD_j: (sq ft) (F), (Calculated in XYZ)
CFN_j: Transfer functions for internal furnishings, Btu per
DFN_j: (hr) (sq ft) (F), (Calculated in XYZ)

where $j = 0, 1, \dots, M$

UG: Heat transmission coefficient of window glass, Btu
per (hr) (sq ft) (F)
CFM: Rate of air flow through the room, cu. ft per min.
(Ventilation rate)

Calculation Sequence:

1. Conversion of the given transfer functions into single series $x_j, y_j,$ and z_j . This calculation is a polynomial division*, i.e.,

$$x_0 z^0 + x_1 z^{-1} + x_2 z^{-2} + x_3 z^{-3} + \dots$$

$$= \frac{a_0 z^0 + a_1 z^{-1} + a_2 z^{-2} + \dots}{1 + b_1 z^{-1} + b_2 z^{-2} + b_3 z^{-3} + \dots}$$

where

$x_0, x_1, x_2 \dots$ = single series response factor set

$a_0, a_1, a_2 \dots$ and

$b_1, b_2, b_3 \dots$ = coefficient of the given numerator

and denominator series respectively

For example, using given notation in this section for the outside wall, the x's, a's and b's are

$x_0 = sCW_0$	$a_0 = CW_0$	$b_0 = 1.0$
$x_1 = sCW_1$	$a_1 = CW_1$	$b_1 = DW_1$
$x_2 = sCW_2$	$a_2 = CW_2$	$b_2 = DW_2$
.	.	$b_3 = DW_3$
.	.	.
.	.	.

* The rules of polynomial division can be obtained from any standard textbooks on numerical analysis.

where the letter "s" in front of CW_j denotes coefficients of the single series

2. Calculation of the single series of Room Air Response Factors, sRMRT. The factors in this series are given by

$$sRMRT_j = AF[sBF_j + sCF_j] + AC[sDC_j + sCC_j] + AP[sBP_j + sCP_j] + AW[sCW_j] + AD[sCD_j] + AFN[sCFN_j] + AG[UG_j]^* + 1.08[CFM_j]^*$$

where $j > 10$ calculate the ratio R_j

$$R_j = \frac{sRMRT_{j+1}}{sRMRT_j}$$

and when $|R_j - R_{j+1}| \leq 0.001$ terminate sRMRT_j calculations.

3. Calculation of RMRT

The calculation of RMRT as a ratio of two series consists of three steps:

- (a) Calculation of denominator, $D(z)$,

$$D(z) = 1.0 - Rz^{-1}$$

where R is the last value of the ratio calculated in the sRMRT_j calculations

* Note that $UG_{j=1} = UG, CFM_{j=1} = CFM, UG_{j>1} = 0.0$ and $CFM_{j>1} = 0.0$.

(b) Calculation of numerator, $N(z)$,

$$N(z) = sRMRT_0 z^0 + (sRMRT_1 - (R)sRMRT_0) z^{-1} \\ + (sRMRT_2 - (R)sRMRT_1) z^{-2} \\ + (sRMRT_3 - (R)sRMRT_2) z^{-3}$$

(c) The RMRT's are then evaluated by equating the following equation to the one in (b) above

$$N(z) = \frac{X_0 + X_1 z^{-1} + X_2 z^{-2}}{Y_0 + Y_1 z^{-1}}$$

Typical values are shown in Table A-16.

Table A-16

WEIGHTING FACTORS FOR THE DEVIATION OF SPACE
TEMPERATURE, RMRT'S*

	Weighting Factor Symbol	Heavy Structure	Medium Structure	Light Structure
MX = 2	X_0	-1.85	-1.81	-1.68
	X_1	+1.95	+1.89	+1.73
	X_2	-0.10	-0.08	-0.05
MY = 1	Y_0	1.00	1.00	1.00
	Y_1	-0.91	-0.87	-0.82

* The X coefficients given in this Table are for a room with zero heat conductance to surrounding spaces and are normalized to unit floor area. To get the X_i coefficients for a room with a total conductance K between room air and surroundings, ventilation rate V_i and infiltration rate VI_i , it is necessary to multiply each X_i value by room floor area and then add $[K + (V_i + VI_i) 1.08] (1.00 - Y_i)$ to the resulting X_0 value (where V_i and VI_i are in cfm and K is in Btu/hr °F).

Note: That X_0 value changes with the changes of V_i and VI_i values.

An Algorithm for Calculating the Rate at Which
Sensible Heat is Extracted From the Space

Data:

SCL_t : Sensible cooling load at time t, which is calculated for
a constant space design temperature of T_M , Btu per hr
(Calculated in HCL)

X_j : Weighting factors for use with θ_{t-j} , for $j = 0, 1, \dots$,
 Y_j : (Calculated in RMRT with typical values shown in Table
A-16)

θ_{t-j} : History of hourly space air temperature deviation from
the assumed constant value T_M , for $j = 1, 2 \dots$, F

C: Average heat extraction rate of the apparatus in a space
when the space air temperature is T_M , Btu per hr

D: Change in the rate of heat extraction of the apparatus
caused by one degree change in space air temperature,
Btu per (hr) (F)

HE_{t-j} : History of heat extracted from the space, for $j = 1, 2$
 \dots , Btu per hr

An Algorithm for Calculating Air Infiltration

It is well recognized that the air infiltration constitutes as much
as 30% of home heating load and a significant part of the load of non-
pressurized commercial buildings. The air leakage of a building depends
upon the tightness of its exterior walls, windows, and doors; the wind
characteristics and temperature difference between the inside and out-
side, and to some extent how the building is operated with respect to
the opening and closing of its door.

The rate of air infiltration can be empirically expressed by

$$Q = C \cdot A \cdot \Delta P^{**N}$$

where

Q: air flow rate

C: flow coefficient

A: flow opening area

N: pressure exponent

ΔP : pressure difference

Unfortunately it is very difficult to determine accurate values of flow
opening area and pressure difference for actual buildings, which consist
of complex air leakage passages. A limited amount of data are given in
the 1972 ASHRAE Handbook of Fundamentals for equivalent opening area of
typical windows, doors and walls. The pressure difference depends upon
the wind characteristics around the building and the temperature differ-

ence between the inside and the outside of the building.

Compiled in this section is a methodology to approximately calculate the pressure difference between a given space and its adjacent space including the outdoor. The basic mathematical principle involved is to attain a solution to a set of pressure difference equations of the following type:

$$Q_i = \sum Q_{i,k} = 0$$

$$Q_{i,k} = \sum A_{i,k} * C_{i,k} * (P_i - P_k)^{**N_{i,k}}$$

where

- Q_i : net air flow out of space i
- $Q_{i,k}$: air exchange between space i and space k
- $A_{i,k}$: flow opening area between space i and k
- $C_{i,k}$: flow coefficient applicable to the air flow between the spaces i and k
- $N_{i,k}$: pressure exponent applicable to the flow between the spaces i and k

A special computational routine is required to solve this set of simultaneous, non-linear equations.

As mentioned previously, air leakage through various openings such as doors, windows, window frames, pinholes in the wall and service shafts may be approximated by an equation of the following type:

$$\begin{aligned} LEAK &= 4000 * A * K * (DP)^{**N} \\ &= C * (DP)^{**N} \end{aligned}$$

where

LEAK = air leakage in cu. ft per min.

A = opening area, sq ft

K = flow coefficient, dimensionless

DP = pressure difference across the opening, inches of water

N = pressure exponent, dimensionless

C = equivalent flow coefficient (EFC)

The values of K and N vary depending upon the type of opening. Moreover, the exact value of A is not well known for many types of openings, such as wall pinholes or cracks around the windows. Table A-17 lists the values of Equivalent Flow Coefficient C and the flow exponent N for various types of openings common to many buildings. These values are derived from the air leakage data compiled in Chapter 19 "Infiltration and Natural Ventilation" of the 1972 ASHRAE Handbook of Fundamentals.

Table A-17

	<u>C</u>	<u>N</u>
1. Double-hung wooden windows (locked)*		
non-weatherstripped loose fit	6	0.66
average fit	2	0.66
weatherstripped loose fit	2	0.66
average fit	1	0.66
2. Window frames*		
masonry frame with no caulking	1.2	0.66
masonry frame with caulking	0.2	0.66
wooden frame	1	0.66
3. Swinging doors* 1/2" crack	160	0.5
1/4" crack	80	0.5
1/8" crack	40	0.5
4. Walls**		
8" plain brick	1	0.8
8" brick and plaster	0.01	0.8
13" brick	0.8	0.8
13" brick and plaster	0.004	0.7
13" brick, furring, lath and plaster	0.03	0.9
frame wall, lath and plaster	0.01	0.55
24" shingles on 1 x 6 boards on 14" center	9	0.66
16" shingles on 1 x 4 boards on 5" center	5	0.66
24" shingles on shiplap	3.6	0.7
16" shingles on shiplap	1.2	0.66

* Values of C listed for these openings are per ft of linear crack length.

** Values of C listed for the walls are per unit area of the wall surface.

In many instances, detailed information of air leakage characteristics is not available, but it is still possible to make a calculation. For a modern office building of 120 ft x 120 ft plan dimension with the floor height of 12 ft, Tamura^{15/} lumped together all the leakage area for a given floor as follows:

Table A-18

outside wall	2.5 sq. ft per story
4 elevator shaft doors	4.5 " " " "
2 stair shaft doors	0.5 " " " "
floor	3.7 " " " "
branch perimeter and interior air duct	7.0 " " " "
return duct	14.0 " " " "
vertical shafts (elevator or stairwell)	1/3 of the cross-sectional area*

The value of C corresponding to these data can be obtained by multiplying them by 2400 which corresponds to $K = 0.6$.

Data:

V: Wind speed measured at a 40 ft elevation as taken from the weather tape, knots

DIR: Wind direction measured clockwise from North, degrees

(see Figure A-21)

* This particular data were derived from a recent and unpublished experiment of the National Bureau of Standards conducted on two high-rise buildings.

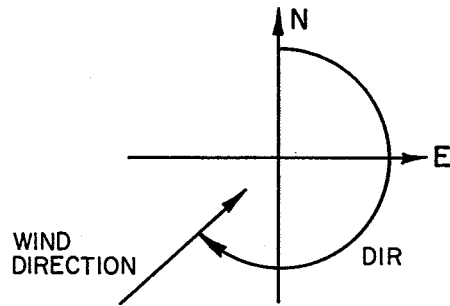


Figure A-21 Definition of Wind Direction Angle

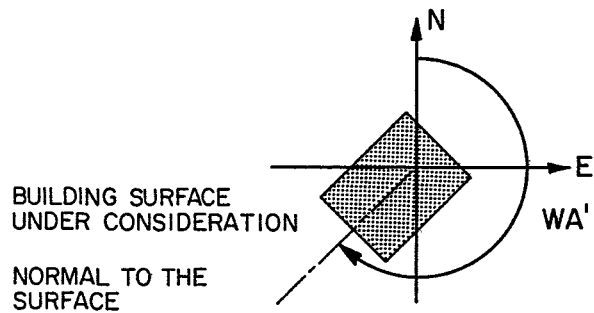


Figure A-22 Definition of the Angle Between North and Normal of the surface Under Consideration

DB: Outdoor air dry-bulb temperature, F

PB: Barometric pressure, in. Hg.

NF: Number of above-grade floors

HTT: Total height of building (from above-grade), ft

TZ: Indoor air temperature, F

TS: Elevator and service shaft temperature

WA': Direction angle of the building as defined with respect to North and the normal of the principal surface of the building (see Figure A-22)

HT_k: Height of the floor, ft, for k = 1, 2, 3, ... NF

CFMSP_k: Ventilation air supplied to the floor, cu ft per min, for k = 1, 2, 3, ... NF

CFMEX_k: Ventilation air exhausted from the floor, cu ft per min, for k = 1, 2, 3, ... NF

Calculation Sequence:

1. $V' = 1.153 * V$

$TO = 460 + DB$

$TI = 460 + TZ$

$PO = 0.4910 * PB$

$x = DIR - WA'$

2. Wind velocity, VH, at height HT on the building, mph

$VH = V' * 0.117 * (1 + 2.81 * \text{Log} (0.305 * HT + 4.75))$

3. Theoretical wind velocity pressure, PTWV on the building,

in. H₂O

$PTWV = 0.000482 * (V ** 2)$

4. Wind direction, BWD, relative to building surfaces

BWD = 1 surface on windward side if,

$$-45^\circ < x < +45^\circ$$

BWD = 2 surface on leeward side if,

$$90^\circ < x < 270^\circ$$

or, $-90^\circ < x < -270^\circ$

BWD = 3 surface on side if,

$$45^\circ < x < 90^\circ$$

or, $-45^\circ < x < 90^\circ$

5. Using Table A-19, determine the normal wind velocity pressure correction factor, PTKN

Table A-19 Values of PTKN

NSB	TB = 1			TB = 2			TB = 3		
	BWD = 1	BWD = 2	BWD = 3	BWD = 1	BWD = 2	BWD = 3	BWD = 1	BWD = 2	BWD = 3
0.5	.1	-.3	-.8	-.5	-.25	-.45	.5	.45	.45
1.0	-.1	-.25	-.5	-.5	-.2	-.3	.45	.3	.3
2.0	.1	-.25	-.4	.0	-.2	-.3	.45	.1	.1
3.0	.1	-.25	-.4	.1	-.2	-.35	.45	.0	.0
5.0	.25	-.35	-.6	.25	-.25	-.45	.5	-.1	-.1
∞	.6	-.35	-.7	.6	-.35	-.7	.6	-.35	-.7

where

TB = 1: Shorter building on windward side

TB = 2: Equals taller building on windward side

TB = 3: Taller building on leeward side

NSB: Ratio of the distance between the adjacent buildings and the width of the building in the direction of wind

6. Wind velocity pressure correction factor, PTKO, for winds obliquely to the wall surface

If BWD = 1 (windward side of building)

$$(PTKO)_m = \text{Cos} (| x |)$$

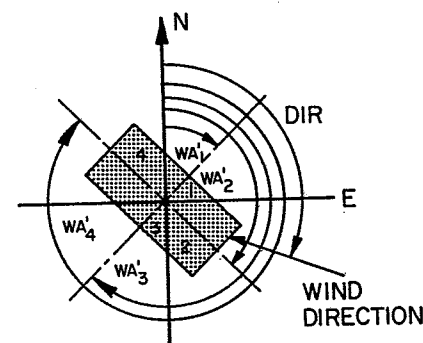
If BWD = 2 (leeward side of building)

$$(PTKO)_1 = 1.0$$

If BWD = 3 (side of building)

$$(PTKO)_s = \text{Cos} (| x |)$$

Example:



DIR = 110°

WA'1 = 45°

WA'2 = 135°

WA'3 = 225°

WA'4 = 315°

Figure A-23 DIR and WA' Angles of Example

Side 1, $DIR-WA_1^I = 110^\circ - 45^\circ = 65^\circ$ (therefore, $BWD = 3$)

Side 2, $DIR-WA_2^I = 110^\circ - 135^\circ = -25^\circ$ (therefore, $BWD = 1$)

Side 3, $DIR-WA_3^I = 110^\circ - 225^\circ = 115^\circ$ (therefore, $BWD = 2$)

Side 4, $DIR-WA_4^I = 110^\circ - 205^\circ = 205^\circ$ (therefore, $BWD = 2$)

Side 1, $(PTKO)_s = \text{Cos } (+65^\circ)$

Side 2, $(PTKO)_m = \text{Cos } (+25^\circ)$

Side 3, $(PTKO)_l = 1.0$

Side 4, $(PTKO)_l = 1.0$

7. Actual wind pressure on the building at height (HT) corresponding to floor (k): $(PAWV)_k$

$$(PAWV)_k = (PTKO)_k * (PTKN)_k * (PTWV),$$

8. Stack effect pressure (PSE) on the outside of the building at building height (HT) and floor (k), in. H_2O

$$(PSE)_k = -0.52 * PO * HT/TO$$

9. Total pressure on the outside of the building (PCO) at floor (k), in. H_2O

$$(PCO)_k = (PAWV)_k + (PSE)_k$$

10. Pressure in the elevator and serve shafts (PSE) at height (HT) corresponding to floor (k), in. H_2O

$$(PSE)_k = -0.52 * PO * HT/TT + (PSE)_1$$

11. Choose appropriate flow coefficients and pressure exponents for air leakage paths of each floor as follows:

Flow coefficients

CWD: Value of C for appropriate window in Table A-17 multiplied by the total crack length of all the windows

CFM: Value of C for appropriate window frame in Table A-17 multiplied by the total crack length of all the window frames

CDR: Value of C for appropriate door in Table A-17 multiplied by the total crack length of all the doors

CWL: Value of C for appropriate walls in Table A-17 multiplied by the total wall area

CCL: Value of A for the ceiling from Table A-18 multiplied by 2400

CFL: Value of A for the floor from Table A-18 multiplied by 2400

CEL: Value of C for elevator doors

CSS: Value of C for the doors to the service shaft

CFS and CES: Value of the cross section of the shaft multiplied by 800

Pressure exponent

NWD: Value of N for the appropriate window in Table A-17

NFM: Value of N for the appropriate window frame in Table A-17

NDR: Value of N for the appropriate door in Table A-17

NWL: Value of N for the appropriate wall in Table A-17

NCL: 0.5

NFL: 0.5

NEL: 0.5

NSS: 0.5

NFS: 0.5

NSE: 0.5

12. Solution of 2 * NF equations

Outdoor air leakage to k-th floor rooms* (see Figure A-23)

Window k leakage

$$LEAKWD_{k,j} = GWD_{k,j} * (PCO_{k,j} - PI_k) ** NWD_{k,j} \quad (1)$$

Window frame leakage

$$LEAKFM_{k,j} = CFM_{k,j} * (PCO_{k,j} - PI_k) ** NFM_{k,j} \quad (2)$$

Door leakage

$$LEAKDR_{k,j} = CDR_{k,j} * (PCO_{k,j} - PI_k) ** NDR_{k,j} \quad (3)$$

Wall leakage

$$LEAKWL_{k,j} = CWL_{k,j} * (PCO_{k,j} - PI_k) ** NWL_{k,j} \quad (4)$$

Ceiling leakage

$$LEAKCL_k = CCL_k * (PI_{k+1} - PI_k) ** NCL_k \quad (5)$$

Floor leakage

$$LEAKFL_k = CFL_k * (PI_{k-1} - PI_k) ** NFL_k \quad (6)$$

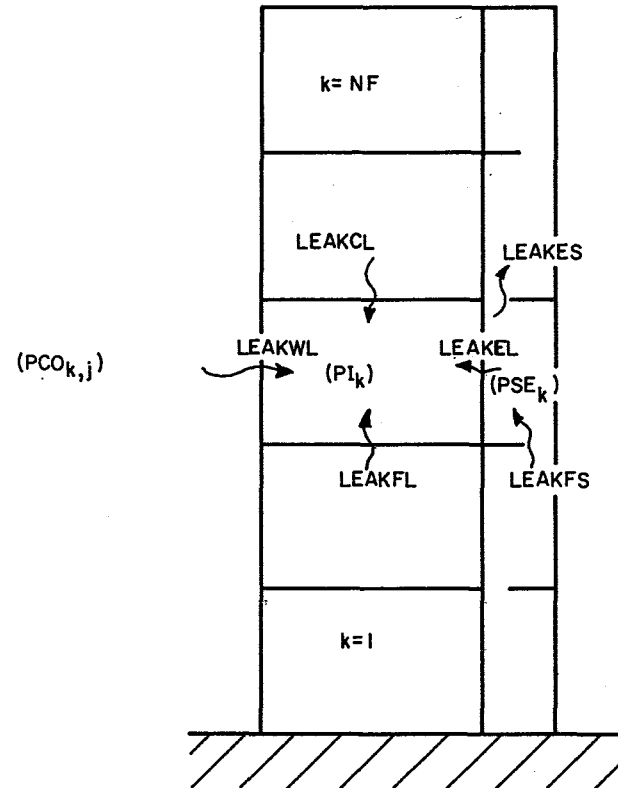


Figure A-23 Air Leakage Pattern of a High-Rise Building

* In all of above expressions, subscript k refers to the k-th floor and subscript j refers to the j-th side of the building where the convention is j = 1 (south), 2 (west), 3 (north), and 4 (east).

Leakage from the elevator and service shafts*

$$\text{LEAKEL}_k = \text{CEL}_k * (\text{PSE}_k - \text{PI}_k) ** \text{NEL}_k \quad (7)$$

$$\text{LEAKSS}_k = \text{CSS}_k * (\text{PSE}_k - \text{PI}_k) ** \text{NSS}_k \quad (8)$$

Air leakage between the floor levels within the shafts*/

$$\text{LEAKFS}_k = \text{CFS}_k * (\text{PSE}_{k-1} - \text{PSE}_k) ** \text{NFS}_k \quad (9)$$

$$\text{LEAKES}_k = \text{CES}_k * (\text{PSE}_{k-1} - \text{PSE}_k) ** \text{NSE}_k \quad (10)$$

In the previous equations, unknowns are PI_k for $k = 1, 2, 3, \dots, \text{NF}$ and PSE_k for $k-1, 2, 3, \dots, \text{NF}$ provided that the pressures in all the shafts are assumed equal at a given floor level.

Flow balance equations at the k -th floor (the individual quantities come from equations 1-10 above)

Rooms

$$\begin{aligned} &\text{LEAKWD}_{k,j} + \text{LEAKFM}_{k,j} + \text{LEAKDR}_{k,j} + \text{LEAKWL}_{k,j} + \text{LEAKCL}_k \\ &+ \text{LEAKFL}_k + \text{LEAKEL}_k + \text{LEAKSS}_k + \text{CFMSP}_k - \text{CFMEX}_k = 0 \end{aligned}$$

Elevator Shaft or Service Shaft

$$\text{LEAKFS}_k + \text{LEAKES}_k - \text{LEAKEL}_k + \text{CFMSPS}_k - \text{CFMEXS}_k = 0$$

where CFMSPS_k : ventilation air supplied at the k -th floor in the shaft

CFMEXS_k : air exhausted from the shaft at the k -th floor

These $2 * \text{NF}$ sets of flow balance equations must be solved by an appropriate iteration technique to obtain the pressure profiles in the building and in the shafts. Then the calculated pressure values are used to determine the air leakage of the building.

Recently a comprehensive computer program that embodies the basic algorithm described in this section was published by D. M. Sander and G. T. Tamura of the National Research Council of Canada. The details of the program are given in an NRC booklet entitled "A Fortran IV Program to Simulate Air Movement in Multi-Storey Buildings", DBR Computer Program No. 35, (March 1973).

* In all of above expressions, subscript k refers to the k -th floor and subscript j refers to the j -th side of the building where the convention is $j = 1$ (south), 2 (west), 3 (north), and 4 (east).

† If this equation were for a service shaft, LEAKEL_k would be replaced by LEAKES_k .

DST

An Algorithm for Determining the Dates of the Daylight Savings Time

Data:

YR: Year AD
MO: Month of the year
DAY: Day of the month

The main variables calculated in this subroutine are:

DSTX: The day when daylight savings time commences
DSTY: The day when standard time resumes
DST: The daylight savings time indicator
0 during the standard time period
DST = 1 during the daylight savings time period

Calculation Sequence:

1. If MO is less than 4 or greater than 10, DST = 0
If MO is greater than 4 and less than 10, DST = 1
2. If MO = 4, DAY is less than 25, DST = 0
If DAY is greater than 23, call WKDAY subroutine
If DAY is Sunday, DSTX = DAY
If DAY is less than DSTX, DST = 0, otherwise DST = 1
3. If MO = 10, DAY is less than 25, DST = 1
If DAY is greater than 24, call WKDAY subroutine
If DAY is Sunday, DSTY = DAY
If DAY is less than DSTY, DST = 1, otherwise DST = 0

WKDAY

An Algorithm Used to Identify the Day of the Week for Any Given Date of the Year From 1901 to 2000

Data:

YR: Year AD
MO: Month of the year
DAY: Day of the month

The variable calculated in this subroutine is WKDAY, the weekday indicator.

1 if Sunday
2 if Monday
3 if Tuesday
WKDAY = 4 if Wednesday
5 if Thursday
6 if Friday
7 if Saturday

Calculation Sequence:

1. Let FSTDAY(1) = 31, FSTDAY(2) = 59, FSTDAY(3) = 90, FSTDAY(4) = 120
FSTDAY(5) = 151, FSTDAY(6) = 181, FSTDAY(7) = 212, FSTDAY(8) = 243
FSTDAY(9) = 273, FSTDAY(10) = 304, FSTDAY(11) = 334, FSTDAY(12) = 365
2. Let N = Integer part of YR/4
ND = N - 485
IY = 2, IADD = 2
If ND = 0, Go to (4)
If ND is less than 0, ND = -ND and IADD = -2

3. Repeat the following steps for ND times

IY = IY - IADD

If IY is greater than 7, IY = IY - 7

If IY is equal to 0, IY = 7

If IY is less than 0, IY = IY + 7

4. Let MD = YR - N * 4

If MD is equal to 0 IWK = IY

If MD is equal to 1 IWK = IY + 2

If MD is equal to 2 IWK = IY + 3

If MD is equal to 3 IWK = IY + 4

If IWK is greater than 7, IWK = IWK - 7

If MO is not equal to 1 go to 5

TDAY = DAY - 1

Go to 7

5. Repeat the following for j = 2 through 12

If MO is equal to j, let TDAY = FSTDAY (j-1) + Day - 1

Otherwise Go to 6

6. If MD is equal to 0 and MO is greater than 2, TDAY = TDAY + 1

7. Let NTX = Integer part of TDAY/7

NDX = TDAY - 7 * NTX + IWK

If NDX is greater than 7, let NDX = NDX - 7

8. Let WKDAY = NDX

9. If this routine is going to be applied for the period outside 1901-2000, the following additional algorithms must be added

KV = First two digits of YR

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KTEST = Last two digits of YR

If MO ≤ 2 or KTEST = 0, KV = KV - 1

LTEST = Remainder of KV/4

KV = 4*LV + LTEST

If LTEST = 2, WKDAY = WKDAY + 1

= 1, WKDAY = WKDAY + 2

= 0, WKDAY = WKDAY + 3

Otherwise WKDAY = WKDAY - 3*(LV - 4)

If WKDAY < 0, WKDAY = WKDAY + 7

If WKDAY > 7, WKDAY = WKDAY - 7

An alternate calculation sequence for WKDAY has been suggested*

***** ALL INTEGER ARITHMETIC *****

1 MDAY=30*(MO-1)+(MO-2)*58/100-1
IF (MO.LE.2) MDAY=MDAY+MO

2 NIVC=IYR-(IYR/400)*400
NLYR=NIVC/4
NCEN=NIVC/100
NIC=NIVC-100*NCEN

3 IY=6-2*NLYR

4 MD=NIVC-NLYR*4
IF (MD.GE.1) IY=IY+1+MD
IY=IY-NCEN
IF (NIC.EQ.0.AND.NIVC.GT.0) IY=IY+1

5 IDAYR=MDAY+IDA

6 IF (NIC.EQ.0.AND.NCEN.GE.1) GO TO 7
IF (MD.EQ.0.AND.MO.GT.2) IDAYR=IDAYR+1

* This was contributed by A. W. Courtney, Scientific Programming, Box 508, Bloomfield Hills, Michigan 48013.

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HOLIDAY

```
7 JWK=IDAYR+IY
  NDX=JWK-(JWK/7)*7
  IF (NDX.LE.Ø) NDX=NDX+7
```

```
8 WKDAY=NDX
```

NOTE: IDA FORMERLY CALLED "DAY"
IYR FORMERLY CALLED "YR"
IDAYR = NUMBER OF THE DAY OF THE YEAR
Ø = NUMERICAL ZERO

An Algorithm to Identify the National Holidays of the
United States of America

Simple modifications allow the identification of any holidays or any special days in any country as long as the Gregorian Calendar system is employed.

Data:

YR: Year AD

MO: Month

DAY: Day of the month

The primary variable calculated in this subroutine is HOL, the holiday indicator; it is 1 if the date is a holiday and zero if it is a non-holiday.

Calculation Sequence:

HOL = 1

If MO = 1 and DAY = 1

MO = 12, DAY = 31 and WKDAY = 6

MO = 1, DAY = 2 and WKDAY = 2

MO = 2, 22 > DAY ≥ 15 and WKDAY = 2

MO = 5, DAY ≥ 25 and WKDAY = 2

MO = 7 and DAY = 4

MO = 7, DAY = 3 and WKDAY = 6

MO = 7, DAY = 5 and WKDAY = 2

MO = 9, 7 > DAY and WKDAY = 2

MO = 10, 15 > DAY ≥ 8 and WKDAY = 2

MO = 10, 29 > DAY > 22 and WKDAY = 2

MO = 11, 29 > DAY > 21 and WKDAY = 5

MO = 12, DAY = 24 and WKDAY = 6

MO = 12, DAY = 26 and WKDAY = 2

otherwise HOL = 0

PSY*

Various Algorithms for Approximate Psychrometric Calculations

The following symbols are used throughout the PSY subroutines:

DB: Dry-bulb temperature, F (determined in CLIMATE)

DP: Dewpoint temperature, F (determined in CLIMATE)

WB: Wet-bulb temperature, F (determined in CLIMATE)

t: Temperature, either DB, WB, or DP, F

PB: Barometric pressure, in. Hg (determined in CLIMATE)

H: Enthalpy of moist air, Btu per lb of dry air

HS: Enthalpy of moist air saturated with water vapor,
Btu per lb of dry air

PV: Partial pressure of water vapor in moist air, in. Hg

PVS: Partial pressure of water vapor in moisture saturated
air, in. Hg

V: Volume of moist air, cu ft per lb of dry air

W: Humidity ratio of moist air, lb of water vapor per
lb dry air

log(x): Natural logarithm of x

log10(x): Common logarithm of x

The following algorithms are used for calculating the psychrometric properties of moist air. All of these are not required for load calculations but are presented here in a package and can be applied in a variety of engineering applications.

a. PVS (t)

1. Let $A(1) = -7.90298$ $B(1) = -9.09718$
 $A(2) = 5.02808$ $B(2) = -3.56654$
 $A(3) = -1.3816 \text{ E-}7$ $B(3) = 0.876793$
 $A(4) = 11.344$ $B(4) = 0.0060273$
 $A(5) = 8.1328 \text{ E-}3$
 $A(6) = -3.49149$

2. Let $T = (t + 459.688)/1.8$
 if T is less than 273.16, go to 3

Otherwise

Let $z = 373.16/T$
 $P1 = A(1) * (z-1)$
 $P2 = A(2) * \log_{10}(z)$
 $P3 = A(3) * (10^{A(4)} * (1-1/z))^{-1}$
 $P4 = A(5) * (10^{A(6)} * (z-1))^{-1}$

Go to 4.

3. Let $z = 273.16/T$
 $P1 = B(1) * (z-1)$
 $P2 = B(2) * \log_{10}(z)$
 $P3 = B(3) * (1-1/z)$
 $P4 = \log_{10}(B(4))$

4. $PVS = 29.921 * (10^{P1 + P2 + P3 + P4})$

* When the exact Goff-Gratch method is required, algorithms described in Reference 17 should be used. Tables A-19 and A-20 taken from that reference list that the psychrometric properties calculated by the PSY routines and the exact Goff-Gratch method, respectively. From examination of these tables it can be seen that the values calculated by the PSY subroutines are in very good agreement with the values calculated by the exact Goff-Gratch method.

b. PV (DB, WB, PB)

1. $PVP = PVS(WB)$
 $WS = 0.622 * PVP / (PB - PVP)$
IF $(WB \leq 32)$ go to 3
 $HL = 1093.049 + 0.441 * DB - WB$
 $CH = 0.24 + 0.441 * WS$
 $WH = WS - CH * (DB - WB) / HL$

2. $PV = PB * WH / (0.622 + WH)$

3. $PV = PVP - 5.704 * 10^{-4} * PB * (DB - WB) / 1.8$

c. W (DB, WB, PB)

1. $VP = PV (DB, WB, PB)$

2. $W = 0.622 + VP / (PB - VP)$

d. H (DB, WB, PB)

$$H = 0.24 * DB + (1061 + 0.444 * DB) * W (DB, WB, PB)$$

e. V (DB, WB, PB)

1. $WV = W (DB, WB, PB)$

2. $V = 0.754 * (DB + 459.7) * (1 + 7000 * WV / 4360) / PB$

f. H (DB, DP, PB)

1. $W = 0.622 * PVS(DP) / (PB - PVS(DP))$

2. $H = 0.24 * DB + (1061 + 0.444 * DB) * W$

g. WB (H, PB)

1. If $PB = 29.92$ and $H > 0$

Let $Y = \log(H)$

For $H < 11.758$

$$WB = 0.6040 + 334.841 * Y + 1.3601 * (Y^{**2}) + 0.9731 * (Y^{**3})$$

For $H > 11.758$

$$WB = 30.9185 - 39.682 * Y + 20.5841 * (Y^{**2}) - 1.758 * (Y^{**3})$$

If $PB \neq 29.92$, or $H \leq 0$ solve the following equation by iterating WB

$$H = 0.24 * WB + (1061 + 0.444 * WB) * W (WB, WB, PB)$$

h. DP (PV)

1. Let $Y = \log(PV)$

If PV is less than 0.18036

$$DP = 71.98 + 24.873 * Y + 0.8927 * (Y^{**2})$$

Otherwise

$$DP = 79.047 + 20.579 * Y + 1.8893 * (Y^{**2})$$

Attached to this section are the Fortran listings of subroutines developed at the National Bureau of Standards which incorporate the psychrometric algorithms described above.

PVSF(X) corresponds to PVS (t)

DPF(PV) corresponds to DP (PV)

WBSF(H,PB) corresponds to WB (H, PB)

The routine entitled PSY1 generates dewpoint temperature, vapor pressure, humidity ratio, enthalpy, specific volume, and relative humidity when the dry-bulb temperature, wet-bulb temperature and the barometric pressure are provided as input. This subroutine essentially combines all the algorithms described in this section. PSY2 is similar to PSY1 except that the dewpoint temperature is given in lieu of the wet-bulb temperature.

```

SUBROUTINE PSY1(DB,WB,PR,DP,PV,W,H,V,RH)
THIS SUBROUTINE CALCULATES VAPOR PRESSURE(PV),HUMIDITY RATIO (W)
ENTHALPY(H),VOLUME(V),RELATIVE HUMIDITY(RH) AND DEW-POINT
TEMPERATURE WHEN THE DRY-BULB TEMPERATURE(DB),WET-BULB TEMPERATUR
(WB) AND BAROMETRIC PRESSURE(PB) ARE GIVEN
PVP=PVSF(WB)
IF(DB-WB)4,4,5
5 WSTAR=0.622*PVP/(PB-PVP)
IF(WB-32.)1,1,2
1 PV=PVP-5.704E-4*PB*(DB-WB)/1.8
GO TO 3
4 PV=PVP
GO TO 3
2 CDB=(DB-32.)/1.8
CWB=(WB-32.)/1.8
HL=597.31+0.4409*CDB-CWB
CH=0.2402+0.4409*WSTAR
EX=(WSTAR-CH*(CDB-CWB)/HL)/0.622
PV=PB*EX/(1.+EX)
3 W=0.622*PV/(PB-PV)
V=0.754*(DB+459.7)*(1+7000*W/4360)/PB
H=0.24*DB+(1061+0.444*DB)*W
DP=DPF(PV)
RH=PV/PVSF(DB)
RETURN
END

```

```

SUBROUTINE PSY2(DB,DP,PR,WB,PV,W,H,V,RH)
THIS SUBROUTINE CALCULATES THE FOLLOWINGS WHEN DRY-BULB TEMPERATURE
(DB),DEW-POINT TEMPERATURE(DP),AND BAROMETRIC PRESSURE(PB) ARE GIVEN
WB WET-BULB TEMPERATURE
W HUMIDITY RATIO
H ENTHALPY
V VOLUME
PV VAPOR PRESSURE
RH RELATIVE HUMIDITY
PV=PVSF(DP)
PVS=PVSF(DB)
RH=PV/PVS
W=0.622*PV/(PB-PV)
V=0.754*(DB+459.7)*(1+7000*W/4360)/PB
H=0.24*DB+(1061+0.444*DB)*W
WB=WB1(H,PB)
RETURN
END

```

```

FUNCTION WBF(H,PB)
THIS PROGRAM APPROXIMATES THE WET-BULB TEMPERATURE WHEN
ENTHALPY AND BAROMETRIC PRESSURE ARE GIVEN

```

```

IF(PB.NF.29.92) GO TO 2
Y=LOG(H)
IF(H.GT.11.7581) GO TO 3
WBF=0.6041+3.4841*Y+1.3601*Y*Y+0.97307*Y*Y*Y
GO TO 4
3 WBF=30.9185-39.68200*Y+20.5841*Y*Y-1.758*Y*Y*Y
GO TO 4
2 WB1=150.
PV1=PVSF(WB1)
W1=0.622*PV1/(PB-PV1)
X1=0.24*WB1+(1061+0.444*WB1)*W1
Y1=H-X1
9 WB2=WB1-1
PV2=PVSF(WB2)
W2=0.622*PV2/(PB-PV2)
X2=0.24*WB2+(1061+0.444*WB2)*W2
Y2=H-X2
IF(Y1*Y2) 6,7,8
8 WB1=WB2
Y1=Y2
GO TO 9
7 IF(Y1) 10,11,10
1 WBF=WB1
GO TO 4
0 WBF=WB2
GO TO 4
6 Z=ABS(Y1/Y2)
WBF=(WB2*Z+WB1)/(1+Z)
4 RETURN
END

```

```

FUNCTION DPF(PV)
THIS SUBROUTINE CALCULATES DEW-POINT TEMPERATURE FOR GIVEN VAPOR PRESSURE
Y=LOG(PV)
IF(PV.GT.0.1836) GO TO 1
DPF=71.98+24.873*Y+0.8927*Y*Y
GO TO 2
1 DPF=79.047+30.579*Y+1.8893*Y*Y
2 RETURN
END

```

```

FUNCTION PVSF(X)
DIMENSION A(6)/-7.90298,5.02808,-1.3816E-7,11.344,8.1328E-3,-3.491
149/,B(4)/-9.09718,-3.56654,0.876793,0.0060273/,P(4)
T=(X+459.688)/1.8
IF(T.LT.273.16) GO TO 3
Z=373.16/T
P(1)=A(1)*(Z-1)
P(2)=A(2)*LOG10(Z)
Z1=A(4)*(1-1/Z)
P(3)=A(3)*(10**Z1-1)
Z1=A(6)*(Z-1)
P(4)=A(5)*(10**Z1-1)
GO TO 4
3 Z=273.16/T
P(1)=B(1)*(Z-1)
P(2)=B(2)*LOG10(Z)
P(3)=B(3)*(1-1/Z)
P(4)=LOG10(B(4))
4 SUM=0
DO 5 I=1,4
5 SUM=SUM+P(I)
PVSF=29.921*10**SUM
RETURN
END

```

Table A-19

PR = 29.92 in. Hg.

	DR	WR	DP	RH (%)	PV	W	H	V
	80.0	80.0	80.0	100.0	1.0323	.02223	43.57	14.09
	80.0	79.0	78.7	95.7	.9883	.02125	42.50	14.06
	80.0	78.0	77.3	91.6	.9453	.02029	41.45	14.04
	80.0	77.0	76.0	87.5	.9031	.01936	40.43	14.02
	80.0	76.0	74.5	83.5	.8619	.01845	39.43	14.00
	80.0	75.0	73.1	79.6	.8215	.01756	38.45	13.98
	80.0	74.0	71.6	75.8	.7820	.01669	37.50	13.97
	80.0	73.0	70.1	72.0	.7433	.01584	36.57	13.95
	80.0	72.0	68.6	68.3	.7054	.01502	35.67	13.93
	80.0	71.0	67.0	64.7	.6682	.01421	34.78	13.91
	80.0	70.0	65.4	61.2	.6319	.01342	33.92	13.89
	80.0	69.0	63.7	57.8	.5963	.01265	33.07	13.88
	80.0	68.0	62.0	54.4	.5615	.01190	32.24	13.86
	80.0	67.0	60.3	51.1	.5273	.01116	31.44	13.84
	80.0	66.0	58.4	47.8	.4939	.01044	30.65	13.83
	80.0	65.0	56.5	44.7	.4611	.00974	29.88	13.81
	80.0	64.0	54.5	41.6	.4290	.00905	29.12	13.80
	80.0	63.0	52.5	38.5	.3976	.00838	28.39	13.78
	80.0	62.0	50.3	35.5	.3668	.00772	27.66	13.77
	80.0	61.0	48.0	32.6	.3366	.00708	26.96	13.76
	80.0	60.0	45.6	29.7	.3070	.00645	26.27	13.74
	80.0	59.0	43.0	26.9	.2780	.00583	25.60	13.73
	80.0	58.0	40.2	24.2	.2495	.00523	24.94	13.71
	80.0	57.0	37.3	21.5	.2216	.00464	24.29	13.70
	80.0	56.0	34.0	18.8	.1943	.00407	23.66	13.69
	80.0	55.0	30.4	16.2	.1675	.00350	23.04	13.68
	80.0	54.0	26.7	13.7	.1412	.00295	22.43	13.67
	80.0	53.0	22.4	11.2	.1154	.00241	21.84	13.65
	80.0	52.0	17.3	8.7	.0900	.00188	21.26	13.64
	80.0	51.0	10.7	6.3	.0652	.00136	20.69	13.63
	80.0	50.0	1.6	4.0	.0408	.00085	20.13	13.62
	80.0	49.0	-14.6	1.6	.0169	.00035	19.59	13.61

Table A-20

Moist Air Properties Calculated by the
Exact Goff-Gratch Method

PB = 29.92 in. Hg

DB	WB	DP	RH (%)	PV	W	H	S	V
80.0	80.0	80.0	100.0	1.0323	.02233	43.69	.0864	14.09
80.0	79.0	78.7	95.7	.9883	.02135	42.61	.0843	14.07
80.0	78.0	77.3	91.6	.9453	.02039	41.56	.0822	14.05
80.0	77.0	76.0	87.5	.9032	.01945	40.53	.0802	14.03
80.0	76.0	74.6	83.5	.8620	.01854	39.53	.0783	14.01
80.0	75.0	73.1	79.6	.8217	.01764	38.55	.0764	13.99
80.0	74.0	71.7	75.8	.7822	.01677	37.60	.0745	13.97
80.0	73.0	70.2	72.0	.7435	.01592	36.67	.0727	13.95
80.0	72.0	68.6	68.4	.7057	.01509	35.76	.0709	13.93
80.0	71.0	67.1	64.8	.6686	.01428	34.87	.0692	13.91
80.0	70.0	65.5	61.2	.6323	.01349	34.00	.0675	13.90
80.0	69.0	63.8	57.8	.5967	.01271	33.15	.0658	13.88
80.0	68.0	62.1	54.4	.5619	.01196	32.32	.0642	13.86
80.0	67.0	60.3	51.1	.5278	.01122	31.51	.0626	13.85
80.0	66.0	58.5	47.9	.4944	.01050	30.72	.0610	13.83
80.0	65.0	56.6	44.7	.4617	.00979	29.95	.0595	13.81
80.0	64.0	54.6	41.6	.4296	.00910	29.19	.0581	13.80
80.0	63.0	52.5	38.6	.3982	.00843	28.45	.0566	13.78
80.0	62.0	50.4	35.6	.3674	.00777	27.73	.0552	13.77
80.0	61.0	48.1	32.7	.3372	.00712	27.02	.0538	13.76
80.0	60.0	45.6	29.8	.3077	.00649	26.33	.0525	13.74
80.0	59.0	43.1	27.0	.2787	.00587	25.66	.0511	13.73
80.0	58.0	40.3	24.2	.2503	.00527	24.99	.0498	13.72
80.0	57.0	37.3	21.5	.2224	.00468	24.35	.0486	13.70
80.0	56.0	34.0	18.9	.1951	.00410	23.71	.0473	13.69
80.0	55.0	30.5	16.3	.1683	.00353	23.09	.0461	13.68
80.0	54.0	26.8	13.8	.1420	.00293	22.49	.0449	13.67
80.0	53.0	22.6	11.3	.1162	.00244	21.89	.0438	13.65
80.0	52.0	17.5	8.8	.0910	.00191	21.31	.0426	13.64
80.0	51.0	11.0	6.4	.0662	.00138	20.74	.0415	13.63
80.0	50.0	2.0	4.1	.0418	.00087	20.18	.0404	13.62

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8. Appendix B

Weighting Factor Method of Calculating
Heating and Cooling Loads and Space Temperature*

* This section was prepared by D. G. Stephenson and G. Mitalas of the National Research Council of Canada for the ASHRAE Task Group on Energy Requirements for Heating and Cooling.

The weighting factor method is based on the assumption that the heat transfer processes occurring in a room can be described by linear equations; and thus that the superposition principle can be used for the calculation of cooling load and space temperature. This means that the relationship between any excitation (e.g., power input to lights) and the corresponding component of the cooling load can be expressed in the form of a characteristic transfer function. Once all the transfer functions have been determined for a room, they can be used to calculate the response to any excitation. The weighting factors are a convenient way of representing these characteristic transfer functions for a room: they relate the Z-transforms of the excitations to the Z-transforms of the corresponding cooling load components.

The Z-transform^{1, 2/}

When a continuous signal, $f(t)$, is sampled at regular intervals of Δ , the output of the sampling device is a train of pulses as shown in Figure B1. The Laplace transform of this output signal is

$$f(0) + f(\Delta)e^{-s\Delta} + f(2\Delta)e^{-2s\Delta} + \dots \quad (1)$$

If Z is substituted for $e^{s\Delta}$, the transform of the output from the sampler is

$$f(0) + f(\Delta)Z^{-1} + f(2\Delta)Z^{-2} + \dots \quad (2)$$

This polynomial in Z^{-1} is the Z-transform of the function $f(t)$. The chief advantage of this type of transform is that it can be obtained just by sampling the function at regular intervals: the successive outputs being the coefficients of successive powers of Z^{-1} in the Z-transform.

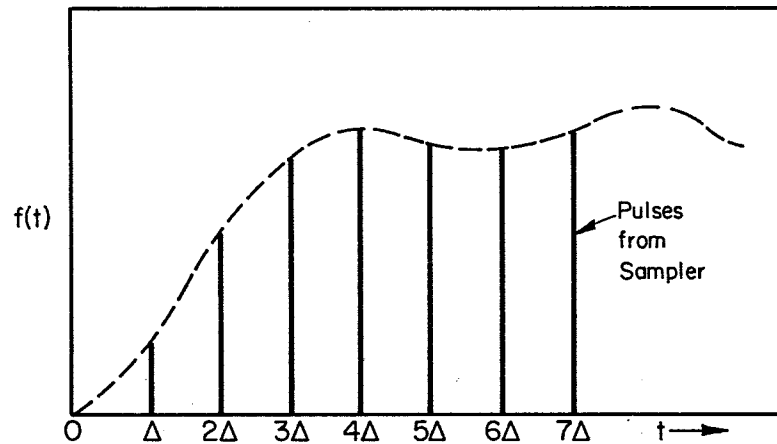
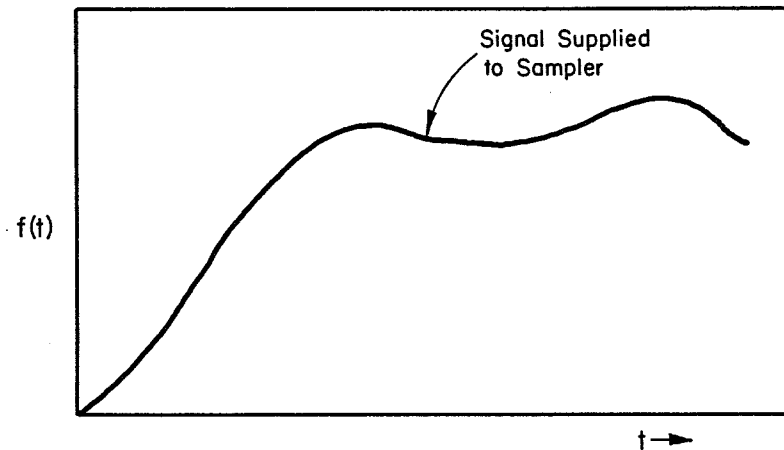


Figure B-1 Pulse Representation of a Continuous Function

If both the input and output of a system are expressed in terms of their Z-transforms, the ratio of the output/input is a Z-transfer function for the system. Assuming that such a transfer function, $K(Z)$, can be found, and that it can be expressed as the ratio of two polynomials in Z^{-1}

$$\text{i.e.,} \quad K(Z) = \frac{a_0 + a_1 Z^{-1} + a_2 Z^{-2} + \dots}{b_0 + b_1 Z^{-1} + b_2 Z^{-2} + \dots} \quad (3)$$

It follows that $O(Z)$, the Z-transform of the output that results from an input represented by $I(Z)$ is

$$O(Z) = K(Z) \cdot I(Z) \quad (4)$$

Both sides of this equation are polynomials so the coefficients of the various powers of Z^{-1} must be the same on the two sides of the equation. Thus, equating the coefficients of Z^{-n} gives

$$O_n \cdot b_0 = I_n \cdot a_0 + I_{n-1} \cdot a_1 + I_{n-2} \cdot a_2 + \dots - \{O_{n-1} \cdot b_1 + O_{n-2} \cdot b_2 + \dots\} \quad (5)$$

where the subscript n on O and I indicates the value of the function at $t = n\Delta$, i.e., O_n is the coefficient of Z^{-n} in the Z-transform $O(Z)$. This expression relates the output at any time $t = n\Delta$ to the input at that time and the values of the input and output at earlier times. The coefficients a_0, a_1, \dots and b_0, b_1, \dots , contain all the characteristics of the system.

Cooling Load, Heat Extraction and Room Temperature

Using the Z-transfer functions approach, the cooling load is the output that results from the input, which is the heat gain. The weighting factor sets are the transfer functions relating the cooling loads to heat gains. The procedure for calculating cooling load is, therefore, first to calculate the various components of the heat gain, and then to combine them with the appropriate weighting factor sets to obtain the cooling load. An expression like equation (5) is used to compute each component of the cooling load.

The cooling load of a space depends on both the magnitude and the nature of its excitations (i.e., outside air temperature, direct and diffuse solar radiation, electric energy input to lights, etc.). The resulting cooling load also depends on the location of the element that absorbs the energy of the excitation. For example, the cooling load profile resulting from one unit of solar radiation absorbed by the window glass is quite different from that of one unit of solar radiation absorbed by the floor surface. To shorten the computation of cooling load, the heat gain must be subdivided into a limited number of components. For example, the total heat gain by a space can be represented by the following components:

- (1) Heat gain through window. (HEATG and HEATG')
- (2) Heat gain through exterior walls and roofs. (HEATX)
- (3) Total power input to lights. (HEATIS)
- (4) Heat gain through doors, partitions, underground walls and floors, and due to internal heat sources other than lights. (HEATDP)

(5) Sensible heat δ_n due to infiltration. (HEATVS)

Each of these heat gain components is calculated on the basis of a constant air temperature in the space. The actual air temperature generally deviates from this reference value, and consequently the rate of heat extraction from the space, HE, differs from the cooling load. The calculation of actual room air temperature and heat extraction rate is the final step in the sequence of calculation: in this case, the previously calculated cooling load is the input along with the characteristics of the air conditioning unit; and heat extraction rate and air temperature are the outputs. If this transfer function is expressed in the form

$$\frac{HE - CL}{\delta} = \sum_0^v x_j z^{-j} / 1 + \sum_1^w y_j z^{-j},$$

or

$$HE_n = CL_n + \sum_0^v x_j * \delta_{n-j} - \sum_1^w y_j * (HE_{n-j} - CL_{n-j}) \quad (6)$$

where δ is the deviation of actual air temperature from the reference value used to calculate the heat gains.

The heat extraction rate given by equation (6) must match the rate given by the characteristic of the air conditioning unit. For example, a cooling unit with a simple proportional control system has a characteristic of the form

$$HE_n = C + D * \delta_n \quad (7)$$

where

C = heat extraction rate of the unit operating in a room
at the reference temperature

D = change in the rate of heat extraction caused by one degree rise in room air temperature

Equations (6) and (7) can be combined to give an explicit expression for δ_n

$$\delta_n = \frac{CL_n - C + \sum_1^v x_j * \delta_{n-j} - \sum_1^w y_j * (HE_{n-j} - CL_{n-j})}{D - x_0} \quad (8)$$

and then equation (7) can be used to evaluate HE_n .

Equation (8) can be used to calculate δ_n even if the cooling equipment is off: when the equipment is not operating, C and D are both zero.

Calculation of Room Weighting Factors

The calculation of the room weighting factors is based on the solution of a set of heat balance equations for all the room air^{3, 4/} (RMTMP). A computer program for evaluating weighting factors has been developed by the National Research Council of Canada^{5/}, based on the procedure given in Reference 4.

Three different groups of room weighting factor sets are computed by this program:

- (1) The first group is very large, consisting of a set of factors for each excitation at each surface.
- (2) The second group combines all of the sets in group 1 that pertain to diffuse solar radiation into a single set; it combines all of the sets for direct solar radiation incident on the room surfaces other than inside window pane, floor, and furniture into another combined set; and lastly, it combines all the sets that

pertain to excitation by the power supplied to the lights into a single set.

- (3) The third group of factors carry the consolidation of the various sets of the limit: there is just one set of factors for each component of heat gain.

It is not intended that the first group should be used directly for room cooling or heating load calculations, as the second group give essentially the same results with considerably less computation^{6/}. The further simplification provided by the use of the third group require that the following assumptions be made:

- (1) That the heat gain through room envelope components can be calculated with sufficient accuracy using combined inside surface heat transfer coefficient and that the room weighting factors are the same for heat gain through window, opaque outside wall, corridor wall and a roof.
- (2) That the fraction of the solar radiation absorbed by the window glass and shade, and transmitted directly into the room as well as the portions absorbed by various room surfaces and furniture are constant during the day.

It is probable that the first assumption will not introduce significant errors; however, the second assumption is questionable. At this time, research information is lacking to establish the possible magnitude of the error introduced by this assumption.

The procedures given in this report to convert room excitation to cooling load and heat extraction are based on the third group of room weighting factors, i.e., RMRG, RMRX, RMRT and RMRTS. In addition, simplified procedures are given for the calculation of the RMRT and RMRTS sets. If the highest possible precision is important, weighting factors in the second group should be used.

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9. Appendix C

NBSLD Data Forms

Recently, the General Services Administration of the federal government has subscribed to a time-sharing system called INFONET, which has in turn made the use of NBSLD practical on a time-sharing system (see figure on page 3c).

Entire subroutines and several main programs, which are parts of NBSLD, have all been placed into INFONET and are now accessible to the various government agencies through teletype terminals.

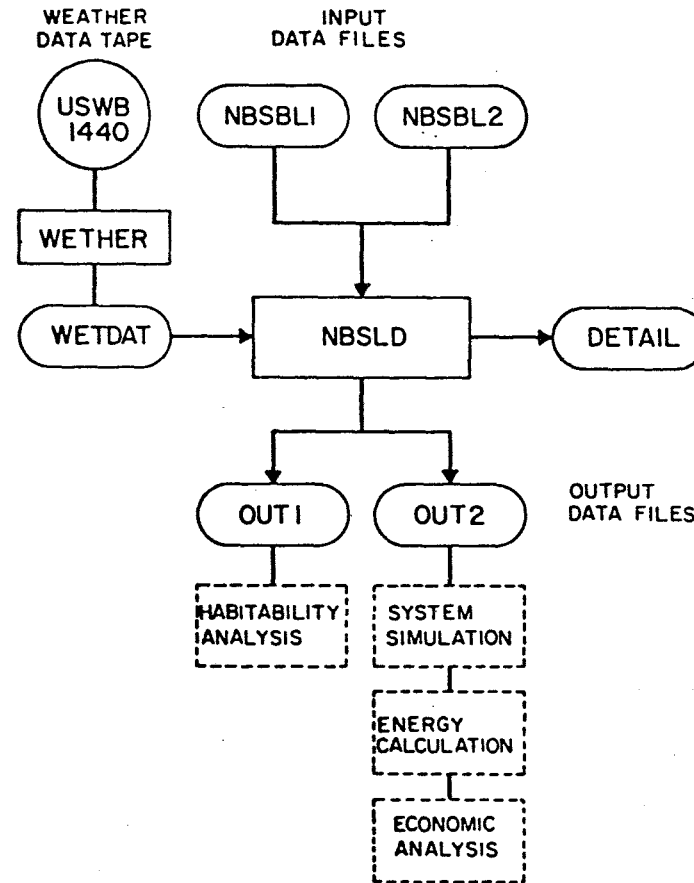
Some of the attractive features of the INFONET system are:

1. Data files for input and output can be created separately from the execution of a main program such that computer time does not have to be sacrificed by the slowness of handling these data. (This is a common problem for heating/cooling load calculation programs.) An input data file can be created, edited and stored for use by a main program, while an output file of intermediate and detailed results can be created in mass storage for later inspection without prolonging the computation time. When a final output is questionable, the analyst can examine the input as well as the intermediate output in detail simply by calling these data files independently.
2. After the execution of a program, the analyst can call for specific values of variables without adding special instructions in the main program to type out these variables.

3. The editing of a program as well as the data files can be made in such a manner that the deletion, addition, or alteration of statements or lines can be made without retyping the entire line.
4. All sub-programs are independently filed, compiled and available to any main program in the same library by an elaborate linking routine. Psychrometric routines, solar heat gain routines, transient heat conduction routines and weather data decoding routines can thus be used by many different main programs.

This time-sharing system is therefore very well suited for design calculations where many different alternatives for building construction such as the area of glass, shading of the windows and thermal mass of the internal structures are to be evaluated as to how they affect heating/cooling loads or energy requirements.

NBSLD on INFONET



A copy of the NBSLD subroutines and main program be obtain through the INFONET system as follows:

```
!U$TDEF,SAVE△RT0105      (△: space)
```

```
!DEVICES,QUEUE△1
```

(Wait for allocation if REQUEST QUEUED by periodically typing

```
!DEVICES)
```

```
!COPY,IN,EVERY△RT0105△-PNC,-REL,-XQT
```

RT0105 is the transportable tape containing all the subroutines and the NBSLD main program, sample data, and sample output.

Sample data file: NBSBL1

Output file: RESULT

The tape also contains the weather decoding routines.

This manual contains the input data forms with instructions for preparing data needed to perform heating and cooling load calculations using NBSLD.

In addition, the manual contains engineering data needed for the computations so that the use of other handbooks or references are generally unnecessary. The required numerical data are to be filled into blank spaces provided on each DATA SHEET and then each sheet can be used directly at a computer terminal or to produce a data card if the program is being run in "batch mode" at a central facility.

General Instruction

Figure C-1 shows the flow diagram or sequence for the data preparation.

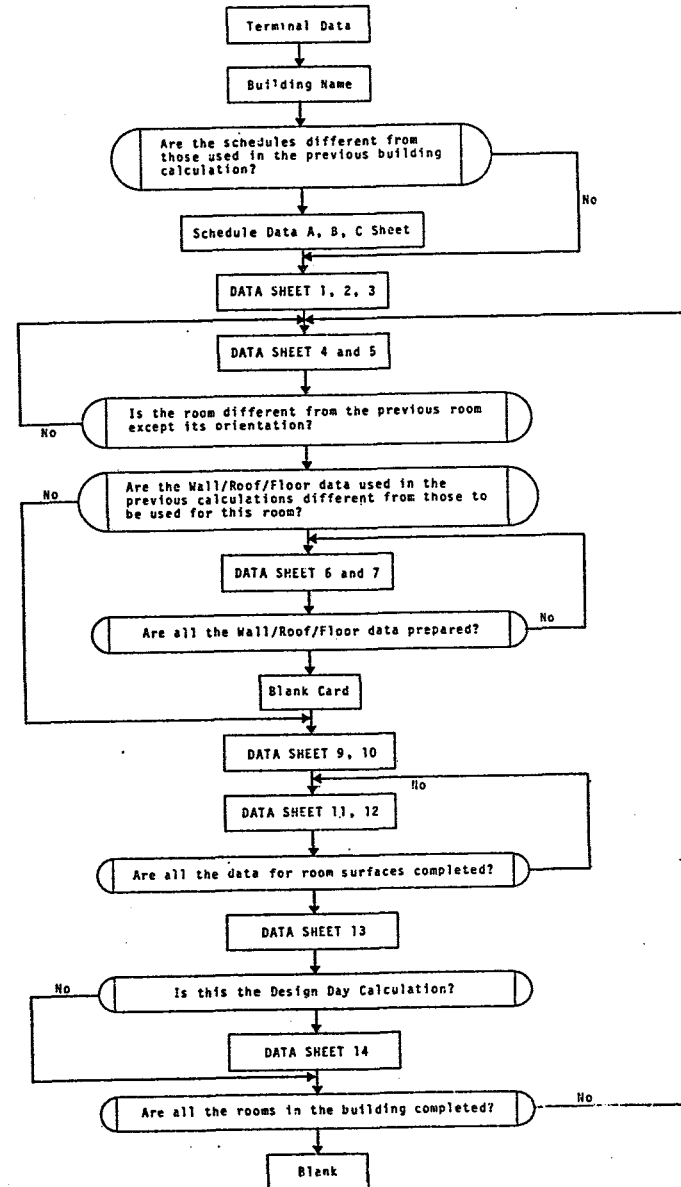
Schedule Profile data sheets A, B, C and D are prepared for a building and are assumed the same for all zones or rooms within the building.

DATA SHEETS 1 through 3 are prepared for each different building, thus need not be repeated for the analysis of rooms within the same building.

DATA SHEETS 6 and 7 usually need to be prepared only for the first room of the building, because other rooms that follow in the same building usually employ the same wall, roof, and floor constructions.

When two rooms have identical shape and construction and differ only in orientation, the second room is considered to be rotated with respect to the first room and requires only DATA SHEETS 4 and 5.

DATA SHEET PREPARATION GUIDE



Operational Data

RUNID: Index for the calculation of Conduction Transfer Functions:

RUNID = 1 if the load calculation being made for this particular run requires the generation of Conduction Transfer Functions for the walls, roof, ceilings and floors in the building. The Conduction Transfer Functions generated during the run will be stored in tape unit 8 for the future rerun.

RUNID = 2 if the Conduction Transfer Functions have already been calculated and stored in tape unit 8 and this particular run does not require the generation of new Conduction Transfer Functions.

RUNTYP: Index for the types of calculations to be performed:

RUNTYP = 1 if the calculation is for the hour by hour determination of heating and cooling load for a specified period by making use of weather data tape (unit 7).

RUNTYP = 2 if the calculation is for the design heating and cooling load. A weather data tape is not required for this run.

ASHRAE: Index for the weighting factor usage:

ASHRAE = 0 if the Weighting Factor Method of the ASHRAE Task Group is replaced by a more exact calculation procedure developed at the National Bureau of Standards.

ASHRAE = 1 if the Weighting Factor Method of the ASHRAE Task Group is used to convert the heat gains and losses to loads. (incomplete)

IDETAL: Index for the output specification:

IDETAL = 0 if output of the run is only the daily maximum and the daily total heating and cooling loads.

IDETAL = 1, output of the run will display input data and details of intermediate results such as Conduction Transfer Functions, Radiation heat exchange factors, solar radiation and solar heat gain.

METHOD: Index for the treatment of room surface radiation heat exchange calculation:

METHOD = 0 if the radiation exchange among the room surfaces are treated individually on room by room basis.

METHOD = 1 if a building zone is treated as a box and all the interior partition walls and floor/ceiling sandwich constructions are treated as a single slab to be distributed uniformly on the floor of the box as an extra layer for that floor. (incomplete)

RFMTAP: Tape drive unit number for the tape to be used by the system simulation program of Ross F. Meriwether. If no tape is needed, RFMTAP = 0.

Schedule Profile-B*

QLITX', QEQUX', QOCUP' are the same as QLITX, QEQUX and QOCUP respectively except that these data are for the weekends.

QLITX'

QEQUX'

QOCUP'

* These data should be all zero for the design day calculation.

Schedule Profile-C*

QLITX'', QEQUX'', QOCUP'' are the same as QLITX, QEQUX, and QOCUP respectively except that these data are for the holidays.

QLITX''

QEQUX''

QOCUP''

* These data should be all zero for the design day calculations.

NDAY: Number of days for which the calculations are to be performed.

NSKIP: Number of days to be skipped in case the computation does not start from the first day of the weather tape, which is usually 00 hour, of January 1 if standard NCC 1440 tape is used.

TAPE 2: Tape unit or file number for the output tape, if 0 an output tape or file is not produced. The output tape or file contains the hourly load and weather data needed for the system simulation.

NDAY	NSKIP	TAPE 2							
------	-------	--------	--	--	--	--	--	--	--

(integer data)

* For the design calculation or when RUNTYP is 2, all three variables listed here can be input as 0.

Month: Month for which the calculation is to be done.

Day: Day of the month on which the calculation is to be done.

ELAPS: Number of days elapsed from January 1 to reach the design day. For example, it is 201 for July 21 of a non-leap year.

DBMAX: Maximum outdoor temperature of the design day for cooling (see Table C1 which has been taken from the 1972 ASHRAE Handbook of Fundamentals).

RANGE: Daily range of the outdoor temperature during the design day, °F, (see Table C1).

WBMAX: Summer design wet-bulb temperature, °F, (see Table C1).

DBMWT: Design outdoor air temperature for the heating load calculation, °F, (see Table C1).

TGS: Summer design ground temperature, °F, (see Table C1).

TCW: Winter design ground temperature, °F, (see Table C1).

* If a weather tape is being used and a conventional design calculation is not being done, all variables listed here can be input as 0 except TGS, TCW and UG.

UG: Ground heat transfer coefficient for design heating load calculations based upon Chapter 21, 1972 ASHRAE Handbook of Fundamentals, Btuh*, (use 0.1 if uncertain).

MONTH	DAY	ELAPS	DBMAX	RANGE	WBMAX	DBMWT	TGS	TGW	UG

LONG: Longitude of the building location, degrees.

LAT: Latitude of the building location, degrees (see Table C1).

TZN: Time zone number:

- 5 Eastern Standard time zone.
- 6 Central Standard time zone.
- 7 Mountain Standard time zone.
- 8 Pacific Standard time zone.

ZLF: Building exterior wall perimeter, ft.*

RHOW: Outdoor air relative humidity for a design heating load calculation, %.

(real variable)

LONG	LAT	TZN	ZLF	RHOW					

* Btuh = Btu per (hr) (sq. ft.) (°F).

* For interior space, use the perimeter bounded by non-air conditioned spaces.

The value of ZLF is needed only when ASHRAE = 1. If the value is unknown use ZLF = 10. For the case ASHRAE = 0 use ZLF = 0.

NAMERM

After this card, the data that follow will refer to this specific room or space in the building.

NAME OF THE ROOM																																											
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44

Start from Column 3

(Alphanumeric data; 34 columns maximum)

IROT: Room rotation index:

= 0: if the room load is to be calculated without reference to any previously described room.

= number of degrees: if this room is to be the same as the previous room except for rotation clockwise a specified number of integer degrees.

ISKIP: Wall/Roof/Floor construction data skip:

= 0: if the room requires a new set of wall/roof/floor construction data.

= 1: if the wall/roof/floor data used by the previous room is reused for this room. If ISKIP = 1, data sheets 6 and 7 should be omitted.

INCLUD: Space load summation index:

= 1: space load not included in the summation.

= 0: space load included in a summation of the space load of the previous room.

(integer variables)

IROT	ISKIP	INCLUD							

Wall/Roof/Floor-A

N: Number of layers of composition in a given wall/roof/floor construction.

By referring to the data of Table C3 (taken from the 1972 ASHRAE Handbook of Fundamentals), give the following information for each of the layers starting from the innermost layer to the Nth layer, which is the outermost layer.

L: Thickness of the layer, ft.

K: Thermal conductivity of the layer, Btuh per (hr) (ft) (°F).

P: Density of the layer material, lb per cu. ft.

C: Specific heat of the layer material, Btu per (lb) (°F).

R: Thermal resistance value of the layer in (hr) (sq. ft.) (°F) per Btu.

For the calculation where the ASHRAE weighting factor method is used (ASHRAE = 1), the innermost layer is always the inside surface thermal resistance. If ASHRAE = 0, omit the inside surface thermal resistance. Outside surface thermal resistance is never used in all cases.

The value of R is to be given only when the layer has no apparent thermal mass. If the value of L, K, P, and C are given, R should be zero. If R is given, L, K, P, and C should all be zero, in particular, L should be zero even if it is physically non-zero. For the ground floor, add a finite thickness slab consisting of a 12" thick earth layer.

Note: (1) At least one of the N layers should have non-zero values of L, K, P, and C.

(2) If two or more consecutive layers have no thermal mass, their thermal resistance values should be combined.

(3) If a particular wall is considered to have no appreciable thermal mass, or if it is desired not to consider the thermal mass effect, data for this particular wall should not be included in the data sheets.

(4) If roof is over attic space the innermost (first) layer should be inside thermal resistance in all cases.

IRF*
1

(integer)

N									
---	--	--	--	--	--	--	--	--	--

(real number)

L	K	P	C	R	
					1st LAYER-innermost
					2nd LAYER
					3rd LAYER
					4th LAYER
					Nth LAYER

* IRF is not the input data in this data sheet. It is the identifier of the particular wall in the sheet.

ARCHGS: Infiltration in terms of number of air changes per hour during the summer* months.

ARCHGW: Infiltration in terms of number of air changes per hour during the winter** months.

ARCHGM: Minimum infiltration in terms of air changes per hour when ARCHGS = 0 and ARCHGW = 0.

ZNORM: Number of rooms of the same type being described in the building.

* June through September.

** October through May.

ROOMNO	QLITY	QEQPX	QCU	FLCG	FRAS	TS	CFMS	ARCHGS	ARCHGW
ARCHGM	ZNORM								

IW: Building weight index:

IW = 1 Heavy structure (approximately 70 lb per sq. ft. of floor use or above).

IW = 2 Medium weight structure (between 30 and 70 lb per sq. ft. of floor area).

IW = 3 Lightweight structure (below 30 lb per sq. ft. of floor area).

IL: Lighting fixture index.

ISTART: Starting hour of occupancy.

ILEAVE: Ending hour of occupancy.

Temperature Control Indices, ITHST and ITK

TUL	TLL	QCMAX	QHMAX	DBVMAX	DBVMIN				

ITHST = 1, ITK = 0: The hourly profile of room temperature, either constant or night setback, is prescribed ... Figure C2.

ITHST = 0, ITK = 1: The room temperature is to be calculated for the room which is neither heated nor cooled ... Figure C3.

ITHST = 1, ITK = 1: The upper and lower limit of the room temperature are given. The room will be heated if the room temperature falls below the lower limit TLL and the room is cooled if the room temperature rises above the upper limit TUL. As long as the room temperature is between these two limits, the room is neither heated nor cooled ... Figure C4.

ITHST = 0, ITK = 0: The same as above except that the maximum capacities of heating and cooling systems are introduced. If the room heating and cooling loads exceed the system heating and cooling capacities respectively, the room temperature drift from the set points TLL and TUL is calculated ... Figure C5.

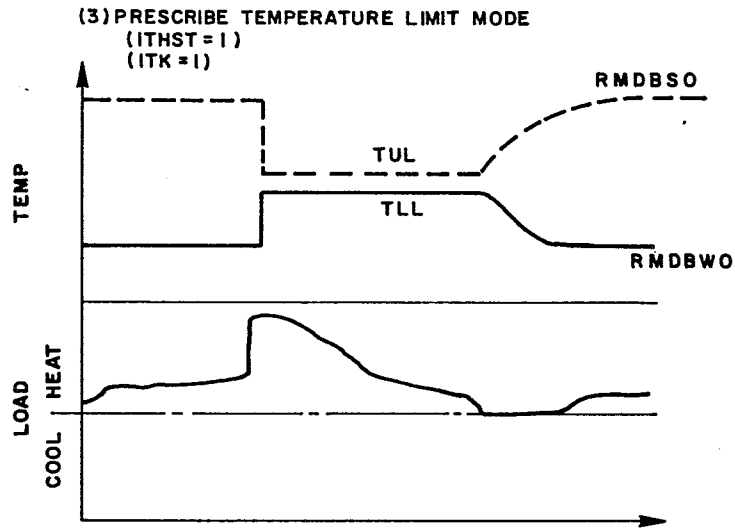


Figure C4

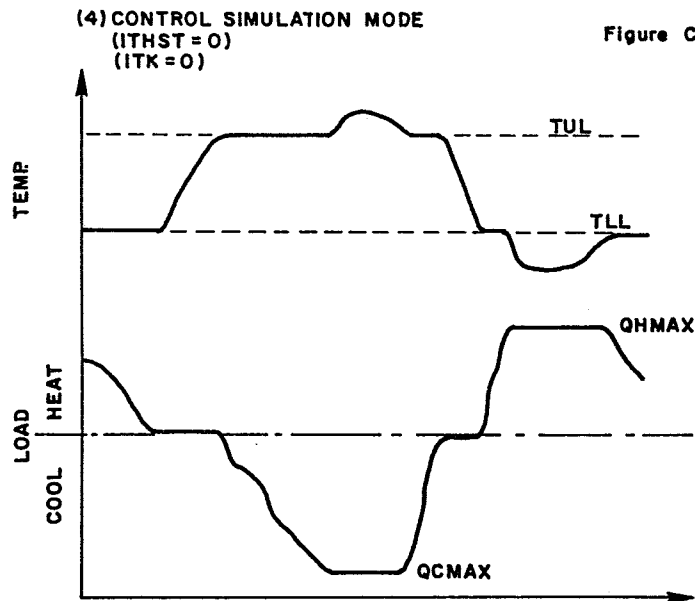
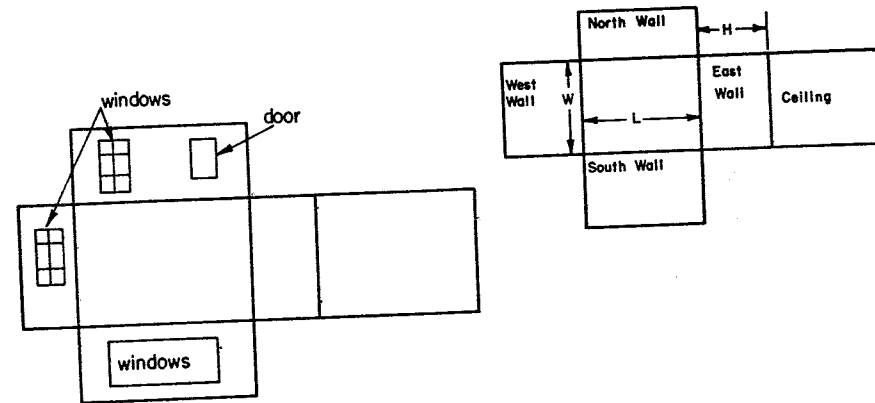


Figure C5

- NS: Number of different type heat transfer surfaces in the south wall.
- NW: Number of different type heat transfer surfaces in the west wall.
- NN: Number of different type heat transfer surfaces in the north wall.
- NE: Number of different type heat transfer surfaces in the east wall.
- L: Length of the room along the south wall, ft.
- W: Width of the room along the west wall, ft.
- H: Height of the room ft.



Note: Number of heat transfer surfaces should not exceed 1 for the ceiling and for the floor.

(integers)*

NS	NW	NN	NE						
L	W	H							

(real values)

* NS + NW + NN + NE should not exceed 28.

ITYPE : Exposure surface type index:

- = 1 if roof.
- = 2 if exterior wall.
- = 3 if window or glass door.
- = 4 if door.
- = 5 if floor on ground or basement wall.
- = 6 if partition walls, party walls, floor/ceiling, furnishings and other internal mass.
- = 7 if completely open.
- = 8 if the adjacent space is not air conditioned and will be considered as having a temperature the same as the outdoors.

IRF: Roof/wall/floor construction identifier index shown in the upper right corner of the roof/wall/floor data sheet 6. If not applicable (such as the cases for lightweight walls, doors and windows), which are not specified in data sheets 6 and 7. IRF = 10.

A: Area of the surface, sq. ft.

AZW: Surface orientation angle, degrees clockwise from south:
 0 for south facing surface, roof/ceiling or floor.
 45 for southwest facing surface.

* See note on data sheet 14.

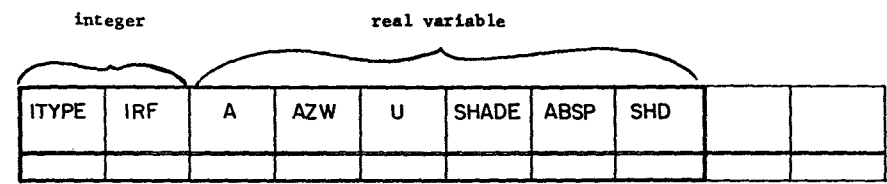
- 90 for west facing surface.
- 135 for northwest facing surface.
- 180 for north facing surface.
- 135 for northeast facing surface.
- 90 for east facing surface.

U: Overall heat transfer coefficient of a surface (Btuh) for which the data for roof/wall/floor are not provided (IRF = 10). For the surface for which roof/wall/floor data are provided (IRF ≠ 10), U should be zero because it will be computed in the program.

SHADE: Shading coefficient for the ITYPE = 3 surface (window or glass door) (see Table C5 which has been taken from the 1972 ASHRAE Handbook of Fundamentals). For all other types of surfaces, this parameter should be zero.

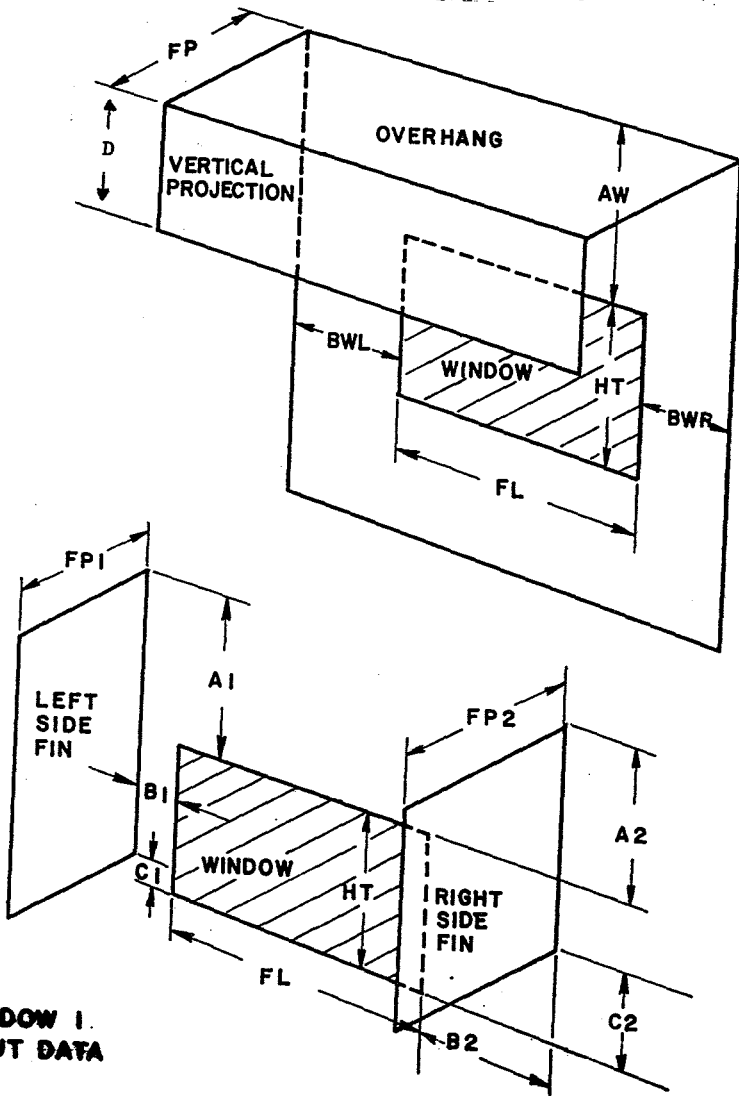
ABSP: Solar absorption coefficient for the exterior surface (see Table C6 which has been taken from Thermal Radiation Properties Survey, G. G. Gubareff, J. E. Jansen, and R. H. Torborg, Honeywell Research Center, 1960). This value should be zero for the surfaces of ITYPE = 3, 5, 6 and 7, and 8.

SHDW: Shadow parameter = 1 if completely shaded by an adjacent building or an external shading device; = 0 if otherwise.



(Exterior Shading Device Data)

Data are given in ft. and as dictated by the figure below.



SHADOW I.
INPUT DATA

50c

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

* The sequence of input should be roof/ceiling, south facing surfaces, west facing surfaces, north facing surfaces, east facing surfaces and floor. While each vertical exposure can accommodate more than one type of surface such as wall, door and window, only one surface should be given for floor and roof/ceiling.

Data Sheets 13 and 14 are to be repeated for each of all the surfaces of the room.

51c

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

Repetition of Data Sheets 13 and 14

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

Repetition of Data Sheets 13 and 14

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

ITYPE	IRF	A	AZW	U	SHADE	ABSP	SHDW		

FL	HT	FP	AW	BWL	BWR	D			

FPI	AI	BI	CI	FP2	A2	B2	C2		

UENDW: Overall heat transfer coefficient of the end walls (gables) of the attic space, Btuh.

UCELNG: Overall heat transfer coefficient of the ceiling under the attic, Btuh.

AENDW: Area of the attic end walls, sq. ft.

ATCHT: Attic space height, ft.

ARCHGA: Air change per hr. for attic.

AIRNT: Nighttime air change multiplier with respect to ARCHGA.

UENDW	UCELNG	AENDW	ATCHT	ARCHGA	AIRNT				

IEXTED: Exterior shading control index:

IEXTED = 1 if the exterior shading device is controlled to cut down the direct solar heat gain.

IEXTED = 0 if the exterior shading device is not controlled.

IEXMS: The month at which the exterior shading device control starts.

IEXME: The month at which the exterior shading device control ends.

NTVNT: Outdoor air change per hour (nearest integer value) during the unoccupied period to precool the building. The ventilation system is assumed on only during the cooling hours when $DB_{MIN} < DB < DB_{MAX}$ (Data Sheet 10).

NVENT: Natural ventilation index

NVENT = 1 if outdoor air is brought in during the occupied period and when the outdoor air temperature is less than TS of Data Sheet 8 to minimize the cooling load.

NVENT = 0 otherwise.

* This data sheet provides information for the attic space with a flat roof. If gabled roof, it must be treated as an equivalent flat roof

Run Sequence

(integer data)

IEXTED	IEXMS	IEXME	NTVNT	NVENT					

The step by step procedure to perform the heating and cooling load calculation by using NBSLD on the INFONET system is as follows:

1. Complete the data forms described in this manual.
2. Check the data for probable errors.
3. Turn the computer terminal on.
4. Dial the computer center and listen to the high-pitched tone.
5. Place the telephone receiver onto the acoustic coupler of the terminal.
6. Hit the key "T".

The computer responds with

"PORT:" Port number

"CENTER:"

Type in after "CENTER:" BB

The computer responds then with

"LOGON:"

Type in your identification number after "LOGON:".

7. The computer then returns the carriage of the terminal and types!
8. Every time the computer waits for your command, it responds with ! at the first position of the carriage. Following in the sequence of the commands needed to perform the load calculations.

! EDIT NBSBLI

↑ 1

2

60

61

↑ Q

SRU'S: .9

! EQUATE 7 WETDAT

! EQUATE 9 SPACE 1

! EQUATE 10 SPACE 2

Type in the data from your data forms
as illustrated in Figure C1.

... all the data are completed.

close the data file.

computer time unit used in the data
preparation.

Weather tape file name.

Output tape No. 1.

Output tape No. 2.

9. Instruction for the terminal data input

Type in the following terminal data:

RUNID, RUNTYP, ASHRAE, IDETAL, METHOD

At this point, the computer starts the load calculation and out-
put such as shown in P103 will be typed out on the terminal.

The use of Weather Data Tape 1440 provided by the National Climatic
Center may be made as follows:

1. Request the tape containing data for specified years from
the National Climatic Center

G. McKay or D. Calloway
Environmental Data Services
Asheville, North Carolina 28801
Telephone: (704) 254-0961

Remember that beginning January 1, 1965 a new program was
initiated for most Weather Bureau Stations reducing the
number of hourly observations being recorded from 24 to
8 per day. This format is not compatible with NBSLD as
it is presently written since it requires hourly weather
data. Note that the tape is 7 channel, 556 BPI density
and of even parity.

2. Have the tape mailed to INFONET computing center at the
following address:

Mr. K. Walls
Center 1
INFONET Division
Computer Science Corporation
650 North Sepulveda Blvd.
El Segundo, California 90245

3. Ask INFONET to assign a Volume number such as US001.
4. The tape 1440 is then decoded and stored into a weather
data file name of which may be obtained by the following
INFONET commands:

```
! DEVICES, QUEUE 0,0,1
```

```
! EQUATE 9 WETDAT
```

```
! WETHER
```

```
VOL, DENS, PARITY, TRACK          Fortran Format
```

```
    VOL: Tape name.                2A4
```

```
    DENS: Tape density in BPI.     A4
```

```
    PARITY: Date parity, either even or odd.  A4
```

```
    TRACK: Track number, either 7 or 9.      A4
```

```
ISKIP, NDAY, IWRITE ... data
```

```
where    ISKIP: Number of days to be skipped from the  
          beginning of the tape.
```

```
          NDAY: Number of days for which the weather  
          data are to be stored in the file.
```

```
          IWRITE = 1: if the weather data are to be printed  
          out on the terminal as they are pro-  
          cessed and stored; otherwise zero.
```

5. The constants of WETDAT may be checked by a separate routine

WETAP by

```
! EQUATE 9 WETDAT
```

```
! WETAP
```

```
NDAY, NSKIP
```

```
where    NDAY: Number of days for which the weather data  
          are to be displayed on the terminal.
```

```
          NSKIP: Number of days to be skipped from the be-  
          ginning of the file WETDAT.
```

When the use is made of WETDAT in any other program, it can be read in the Fortran program as follows:

```
    READ(9) DB, DP, WB, WS, PB, TC, NTOC, DAY, IYEAR, MONTH, ICITY  
where DB, DD, WB, WS, PB, TC, and NTOC are all dimensioned 24  
and represent respectively dry-bulb temperature, dewpoint tem-  
perature, wet-bulb temperature, wind speed, barometric pressure,  
total cloud amount, and type of cloud.
```

Climatic Conditions for United States and Canada*^a

Col. 1 State and Station ^b	Col. 2 Latitude ^c	Col. 3 Elev., ft	Winter				Summer						
			Median of Annual Ex- tremes	Col. 4		Col. 5 Coinci- dent Wind Ve- locity ^d	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range ^e	Col. 8 Design Wet-Bulb		
				99%	97½%		1%	2½%	5%		1%	2½%	5%
ALABAMA													
Alexander City.....	33 0	660	12	16	20	L	96	94	93	21	79	78	77
Anniston AP.....	33 4	599	12	17	19	L	96	94	93	21	79	78	77
Auburn.....	32 4	730	17	21	25	L	98	96	95	21	80	79	78
Birmingham AP.....	33 3	610	14	19	22	L	97	94	93	21	79	78	77
Decatur.....	34 4	580	10	15	19	L	97	95	94	22	79	78	77
DOTHAN AP.....													
Dothan AP.....	31 2	321	19	23	27	L	97	95	94	20	81	80	79
Florence AP.....	34 5	528	8	13	17	L	97	95	94	22	79	78	77
Gadsden.....	34 0	570	11	16	20	L	96	94	93	22	78	77	76
Huntsville AP.....	34 4	619	-6	13	17	L	97	95	94	23	78	77	76
Mobile AP.....	30 4	211	21	26	29	M	95	93	91	18	80	79	79
MOBILE CO.....													
Mobile CO.....	30 4	119	24	28	32	M	96	94	93	16	80	79	79
Montgomery AP.....	32 2	195	18	22	26	L	98	95	93	21	80	79	78
Selma-Craig AFB.....	32 2	207	18	23	27	L	98	96	94	21	81	80	79
Talladega.....	33 3	565	11	15	19	L	97	95	94	21	79	78	77
Tuscaloosa AP.....	33 1	170 ^f	14	19	23	L	98	96	95	22	81	80	79
ALASKA													
Anchorage AP.....	61 1	90	-29	-25	-20	VL	73	70	67	15	63	61	59
Barrow.....	71 2	22	-49	-45	-42	M	58	54	50	12	54	51	48
Fairbanks AP.....	64 5	436	-59	-53	-50	VL	82	78	75	24	64	63	61
Juneau AP.....	58 2	17	-11	-7	-4	L	75	71	68	15	66	64	62
Kodiak.....	57 3	21	4	8	12	M	71	66	63	10	62	60	58
Nome AP.....	64 3	13	-37	-32	-28	L	66	62	59	10	58	56	54
ARIZONA													
Douglas AP.....	31 3	4698	13	18	22	VL	100	98	96	31	70	69	68
Flagstaff AP.....	35 1	6973	-10	0	5	VL	84	82	80	31	61	60	59
Fort Huachuca AP.....	31 3	4664	18	25	28	VL	95	93	91	27	69	68	67
Kingman AP.....	35 2	3446	18	25	29	VL	103	100	97	30	70	69	68
Nogales.....	31 2	3800	15	20	24	VL	100	98	96	31	72	71	70
PHOENIX AP.....													
Phoenix AP.....	33 3	1117	25	31	34	VL	108	106	104	27	77	76	75
Prescott AP.....	34 4	5014	7	15	19	VL	96	94	91	30	67	66	65
Tucson AP.....	33 1	2584	23	29	32	VL	105	102	100	26	74	73	72
Winslow AP.....	35 0	4880	2	9	13	VL	97	95	92	32	66	65	64
Yuma AP.....	32 4	199	32	37	40	VL	111	109	107	27	79	78	77
ARKANSAS													
Blytheville AFB.....	36 0	264	6	12	17	L	98	96	93	21	80	79	78
Camden.....	33 4	116	13	19	23	L	99	97	96	21	81	80	79
El Dorado AP.....	33 1	252	13	19	23	L	98	96	95	21	81	80	79
Fayetteville AP.....	36 0	1253	3	9	13	M	97	95	93	23	77	76	75
Fort Smith AP.....	35 2	449	9	15	19	M	101	99	96	24	79	78	77
HOT SPRINGS NAT. PK.....													
Hot Springs Nat. Pk.....	34 3	710	12	18	22	M	99	97	96	22	79	78	77
Jonesboro.....	35 5	345	8	14	18	M	98	96	95	21	80	79	78
Little Rock AP.....	34 4	257	13	19	23	M	99	96	94	22	80	79	78
Fine Bluff AP.....	34 1	204	14	20	24	L	99	96	95	22	81	80	79
Texarkana AP.....	33 3	361	16	22	26	M	99	97	96	21	80	79	78
CALIFORNIA													
Bakersfield AP.....	35 2	495	26	31	33	VL	103	101	99	32	72	71	70
Barstow AP.....	34 5	2142	18	24	28	VL	104	102	99	37	73	72	71
Blythe AP.....	33 4	390	26	31	35	VL	111	109	106	28	78	77	76
Burbank AP.....	34 1	699	30	36	38	VL	97	94	91	25	72	70	69
Chico.....	39 5	205	23	29	33	VL	102	100	97	36	71	70	69
Concord.....	38 0	195	27	32	36	VL	96	92	88	32	69	67	66

* Data for U. S. stations extracted from *Evaluated Weather Data for Cooling Equipment Design, Addendum No. 1, Winter and Summer Data*, with the permission of the publisher, Union Products Company, Inc., Box 1267, Santa Rosa, California.
^a Data compiled from official weather stations, where hourly weather observations are made by trained observers, and from other sources. Table 1 prepared by ASHRAE Technical Committee 2.2, Weather Data and Design Conditions. Percentage of winter design data show the percent of 3-month period, December through February. Canadian data are based on January only. Percentage of summer design data show the percent of 4-month period, June through September. Canadian data are based on July only. Also see References 1 to 7.
^b When airport temperature observations were used to develop design data, "AP" follows station name, and "AFB" follows Air Force Bases. Data for stations followed by "CO" came from office locations within an urban area and generally reflect an influence of the surrounding area. Stations without designation can be considered rural and may be directly compared with most airport data.
^c Latitude is given to the nearest 10 minutes, for use in calculating solar loads. For example, the latitude for Anniston, Alabama is given as 33 4, or 33°40'.
^d Elevations are ground elevations for each station as of 1964. Temperature readings are generally made at an elevation of 5 ft above ground, except for locations marked "i", indicating roof exposure of thermometer.
^e Coincident wind velocities derived from approximately coldest 600 hours out of 20,000 hours of December through February data per station. Also see References 3 and 6. The four classifications are:
 VL = Very Light, 70 percent or more of cold extreme hours ≤ 7 mph. M = Moderate, 50 to 74 percent cold extreme hours > 7 mph.
 L = Light, 50 to 69 percent cold extreme hours ≤ 7 mph. H = High, 75 percent or more cold extreme hours > 7 mph, and 50 percent are > 12 mph.
 The difference between the average maximum and average minimum temperatures during the warmest month.
^f More detailed data on Arizona, California, and Nevada may be found in *Recommended Design Temperatures, Northern California*, published by the Golden Gate Chapter; and *Recommended Design Temperatures, Southern California, Arizona, Nevada*, published by the Southern California Chapter.

Table C1

Design Weather Data

Reprinted by permission from the 1972 Handbook of Fundamentals, (American Society of Heating, Refrigerating and Air-Conditioning Engineers, 345 East 47th Street, New York), pp. 669-687.

Climatic Conditions for United States and Canada (Continued)*-a

Col. 1 State and Station ^b	Col. 2 Latitude ^c	Col. 3 Elev. ^d ft	Winter			Summer								
			Col. 4		Col. 5 Coincident Wind Velocity ^e	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range ^f			Col. 8 Design Wet-Bulb		
			Median of Annual Ex- tremes	99%		97½%	1%	2½%	5%	1%	2½%	5%		
													99%	97½%
CALIFORNIA (continued)														
Covina	34 0	575	32	38	41	VL	100	97	94	31	73	72	71	
Crescent City AP	41 5	50	26	33	36	L	72	69	65	18	61	60	59	
Downey	34 0	116	30	35	38	VL	93	90	87	22	72	71	70	
El Cajon	32 4	525	26	31	34	VL	98	95	92	30	74	73	72	
El Centro AP	32 5	-30	26	31	35	VL	111	109	106	34	81	80	79	
Escondido	33 0	660	28	33	36	VL	95	92	89	30	73	72	71	
Eureka/Arcata AP	41 0	217	27	32	35	L	67	65	63	11	60	59	58	
Fairfield-Travis AFB	38 2	72	26	32	34	VL	98	94	90	34	71	69	67	
Fresno AP	36 5	326	25	28	31	VL	101	99	97	34	73	72	71	
Hamilton AFB	38 0	3	28	33	35	VL	89	85	81	28	71	68	66	
Laguna Beach	33 3	35	32	37	39	VL	83	80	77	18	69	68	67	
Livermore	37 4	545	23	28	30	VL	99	97	94	24	70	69	68	
Lompoc, Vandenberg AFB	34 4	552	32	36	38	VL	82	79	76	20	65	63	61	
Long Beach AP	33 5	34	31	36	38	VL	87	84	81	22	72	70	69	
Los Angeles AP	34 0	99	36	41	43	VL	86	83	80	15	69	68	67	
Los Angeles CO	34 0	312	38	42	44	VL	94	90	87	20	72	70	69	
Merced-Castle AFB	37 2	178	24	30	32	VL	102	99	96	36	73	72	70	
Modesto	37 4	91	26	32	36	VL	101	98	96	36	72	71	70	
Monterey	36 4	38	29	34	37	VL	82	79	76	20	64	63	61	
Napa	38 2	16	26	31	34	VL	94	92	89	30	69	68	67	
Needles AP	34 5	913	27	33	37	VL	112	110	107	27	76	75	74	
Oakland AP	37 4	3	30	35	37	VL	85	81	77	19	65	63	62	
Oceanside	33 1	30	33	38	40	VL	84	81	78	13	69	68	67	
Ontario	34 0	995	26	32	34	VL	100	97	94	36	72	71	70	
Oxnard AFB	34 1	43	32	35	37	VL	84	80	78	19	70	69	67	
Palmdale AP	34 4	2517	18	24	27	VL	103	101	98	35	70	68	67	
Palm Springs	33 5	411	27	32	36	VL	110	108	105	35	79	78	77	
Pasadena	34 1	864	31	36	39	VL	96	93	90	29	72	70	69	
Petaluma	38 1	27	24	29	32	VL	94	90	87	31	70	68	67	
Pomona CO	34 0	871	26	31	34	VL	99	96	93	36	73	72	71	
Redding AP	40 3	495	25	31	35	VL	103	101	98	32	70	69	67	
Redlands	34 0	1318	28	34	37	VL	99	96	93	33	72	71	70	
Richmond	36 0	55	28	35	38	VL	85	81	77	17	66	64	63	
Riverside-March AFB	33 5	1511	26	32	34	VL	99	96	94	37	72	71	69	
Sacramento AP	38 3	17	24	30	32	VL	100	97	94	36	72	70	69	
Salinas AP	36 4	74	27	32	35	VL	87	85	82	24	67	65	64	
San Bernardino, Norton AFB	34 1	1125	26	31	33	VL	101	98	96	38	75	73	71	
San Diego AP	32 4	19	38	42	44	VL	86	83	80	12	71	70	68	
San Fernando	34 1	977	29	34	37	VL	100	97	94	38	73	72	71	
San Francisco AP	37 4	8	32	35	37	L	83	79	75	20	65	63	62	
San Francisco CO	37 5	52	38	42	44	VL	80	77	73	14	64	62	61	
San Jose AP	37 2	707	30	34	36	VL	90	88	85	26	69	67	65	
San Luis Obispo	35 2	315	30	35	37	VL	89	85	82	28	65	64	63	
Santa Ana AP	33 4	1157	28	33	36	VL	92	89	86	28	72	71	70	
Santa Barbara CO	34 3	100	30	34	36	VL	87	84	81	24	67	66	65	
Santa Cruz	37 0	125	28	32	34	VL	87	84	80	28	66	65	63	
Santa Maria AP	34 5	238	28	32	34	VL	85	82	79	23	65	64	63	
Santa Monica CO	34 0	57	38	43	45	VL	80	77	74	16	69	68	67	
Santa Paula	34 2	263	25	33	36	VL	91	89	86	36	72	71	70	
Santa Rosa	38 3	167	24	29	32	VL	95	93	90	34	70	68	67	
Stockton AP	37 5	28	25	30	34	VL	101	98	96	37	72	70	69	
Ukiah	39 1	620	22	27	30	VL	98	96	93	40	70	69	67	
Visalia	36 2	354	26	32	36	VL	102	100	97	38	73	72	70	
Yreka	41 4	2625	7	13	17	VL	96	94	91	38	68	66	65	
Yuba City	39 1	70	24	30	34	VL	102	100	97	36	71	70	69	
COLORADO														
Alamosa AP	37 3	7536	-26	-17	-13	L	84	82	79	35	62	61	60	
Boulder	40 0	5385	-5	-4	8	L	92	90	87	27	64	63	62	
Colorado Springs AP	38 5	6173	-9	-1	4	L	90	88	86	30	63	62	61	
Denver AP	39 5	5283	-9	-2	3	L	92	90	89	28	65	64	63	
Durango	37 1	6550	-10	0	4	VL	88	86	83	30	64	63	62	

Climatic Conditions for United States and Canada (Continued)*-a

Col. 1 State and Station ^b	Col. 2 Latitude ^c	Col. 3 Elev. ^d ft	Winter			Summer								
			Col. 4		Col. 5 Coincident Wind Velocity ^e	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range ^f			Col. 8 Design Wet-Bulb		
			Median of Annual Ex- tremes	99%		97½%	1%	2½%	5%	1%	2½%	5%		
													99%	97½%
COLORADO (continued)														
Fort Collins	40 4	5001	-18	-9	-5	L	91	89	86	28	63	62	61	
Grand Junction AP	39 1	4849	-2	8	11	VL	96	94	92	29	64	63	62	
Greeley	40 3	4648	-18	-9	-5	L	94	92	89	29	65	64	63	
La Junta AP	38 0	4188	-14	-6	-2	M	97	95	93	31	72	71	69	
Leadville	39 2	10177	-18	-9	-4	VL	76	73	70	30	56	55	54	
Pueblo AP	38 2	4639	-14	-5	-1	L	96	94	92	31	68	67	66	
Sterling	40 4	3989	-15	-6	-2	M	95	93	90	30	67	66	65	
Trinidad AP	37 2	5746	-9	1	5	L	93	91	89	32	66	65	64	
CONNECTICUT														
Bridgeport AP	41 1	7	-1	4	8	M	90	88	85	18	77	76	75	
Hartford, Brainard Field	41 5	15	-4	1	5	M	90	88	85	22	77	76	74	
New Haven AP	41 2	6	0	5	9	H	88	86	83	17	77	76	75	
New London	41 2	60	0	4	8	H	89	86	83	16	77	76	75	
Norwalk	41 1	37	-5	0	4	M	91	89	86	19	77	76	75	
Norwich	41 3	20	-7	-2	2	M	88	86	83	18	77	76	75	
Waterbury	41 3	605	-5	0	4	M	90	88	85	21	77	76	75	
Windsor Locks, Bradley Field	42 0	169	-7	-2	2	M	90	88	85	22	76	75	73	
DELAWARE														
Dover AFB	39 0	38	8	13	15	M	93	90	88	18	79	78	77	
Wilmington AP	39 4	78	6	12	15	M	93	90	87	20	79	78	76	
DISTRICT OF COLUMBIA														
Andrews AFB	38 5	279	9	13	16	M	94	91	88	18	79	77	76	
Washington National AP	38 5	14	12	16	19	M	94	92	90	18	78	77	76	
FLORIDA														
Belle Glade	26 4	16	31	35	39	M	93	91	90	16	80	79	79	
Cape Kennedy AP	28 1	16	33	37	40	L	90	89	88	15	81	80	79	
Daytona Beach AP	29 1	31	28	32	36	L	94	92	91	15	81	80	79	
Fort Lauderdale	26 0	13	37	41	45	M	91	90	89	15	81	80	79	
Fort Myers AP	26 4	13	34	38	42	M	94	92	91	18	80	80	79	
Fort Pierce	27 3	10	33	37	41	M	93	91	90	15	81	80	79	
Gainesville AP	29 4	155	24	28	32	L	96	94	93	18	80	79	79	
Jacksonville AP	30 3	24	26	29	32	L	96	94	92	19	80	79	79	
Key West AP	24 3	6	50	55	58	M	90	89	88	9	80	79	78	
Lakeland CO	28 0	214	31	35	39	M	95	93	91	17	80	79	78	
Miami AP	25 5	7	39	44	47	M	92	90	89	15	80	79	79	
Miami Beach CO	25 5	9	40	45	48	M	91	89	88	10	80	79	79	
Ocala	29 1	86	25	29	33	L	96	94	93	18	80	79	78	
Orlando AP	28 3	1067	29	33	37	L	96	94	93	17	80	79	78	
Panama City, Tyndall AFB	30 0	22	28	32	35	M	92	91	90	14	81	80	80	
Pensacola CO	30 3	13	25	29	32	M	92	90	89	14	82	81	80	
St. Augustine	29 5	15	27	31	35	L	94	92	90	16	81	80	79	
St. Petersburg	28 0	35	35	39	42	M	93	91	90	16	81	80	79	
Sanford	28 5	14	29	33	37	L	95	93	92	17	80	79	79	
Sarasota	27 2	30	31	35	39	M	93	91	90	17	80	80	79	
Tallahassee AP	30 2	58	21	25	29	L	96	94	93	19	80	79	79	
Tampa AP	28 0	19	32	36	39	M	92	91	90	17	81	80	79	
West Palm Beach AP	26 4	15	36	40	44	M	92	91	90	16	81	80	80	
GEORGIA														
Albany, Turner AFB	31 3	224	21	26	30	L	98	96	94	20	80	79	78	
Americus	32 0	476	18	22	25	L	98	96	93	20	80	79	78	
Athens	34 0	700	12	17	21	L	96	94	91	21	78	77	76	
Atlanta AP	33 4	1005	14	18	23	H	95	92	90	19	78	77	76	
Augusta AP	33 2	143	17	20	23	L	98							

Climatic Conditions for United States and Canada (Continued)*.a

Col. 1 State and Station ^b	Col. 2 Latitude ^c	Col. 3 Elev. ^d Ft	Winter			Col. 5 Coincident Wind Velocity ^e	Summer						
			Median of Annual Ex- tremes	Col. 4			Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range ^f	Col. 8 Design Wet-Bulb		
				99%	97½%		1%	2½%	5%		1%	2½%	5%
GEORGIA (continued)													
Moultrie	31 1	340	22	26	30	L	97	95	93	20	80	79	78
Rome AP	34 2	637	11	16	20	L	97	95	93	23	78	77	76
Savannah-Travis AP	32 1	52	21	24	27	L	96	94	92	20	81	80	79
Valdosta-Moody AFB	31 0	239	24	28	31	L	96	94	92	20	80	79	78
Waycross	31 2	140	20	24	28	L	97	95	93	20	80	79	78
HAWAII													
Hilo AP	19 4	31	56	50	61	L	85	83	82	15	74	73	72
Honolulu AP	21 2	7	58	60	62	L	87	85	84	12	75	74	73
Kaneohe	21 2	198	58	60	61	L	85	83	82	12	74	73	73
Wahiawa	21 3	215	57	59	61	L	86	84	83	14	75	74	73
IDAHO													
Boise AP	43 3	2842	0	4	10	L	96	93	91	31	68	66	65
Burley	42 3	4180	-5	4	8	VL	95	93	89	35	68	66	64
Coeur d'Alene AP	47 5	2373	-4	2	7	VL	94	91	88	31	66	65	63
Idaho Falls AP	43 3	4730r	-17	-12	-6	VL	91	88	85	38	65	64	62
Leviston AP	46 2	1413	1	6	12	VL	98	96	93	32	67	66	65
Moscow	46 4	2060	-11	-3	1	VL	91	89	86	32	64	63	61
Mountain Home AFB	43 0	2992	-3	2	9	L	99	96	93	36	68	66	64
Pocatello AP	43 0	4444	-12	-8	-2	VL	94	91	88	35	65	63	62
Twin Falls AP	42 3	4148	-5	4	8	L	96	94	91	34	66	64	63
ILLINOIS													
Aurora	41 5	744	-13	-7	-3	M	93	91	88	20	78	77	75
Belleville, Scott AFB	38 3	447	-7	-6	10	M	97	95	92	21	79	78	77
Bloomington	40 3	775	-7	-1	3	M	94	92	89	21	79	78	77
Carbondale	37 5	380	1	7	11	M	98	96	94	21	80	79	78
Champaign/Urbana	40 0	743	-6	0	4	M	96	94	91	21	79	78	77
Chicago, Midway AP	41 5	610	-7	-4	1	M	95	92	89	20	78	76	75
Chicago, O'Hare AP	42 0	658	-9	-4	0	M	93	90	87	20	77	75	74
Chicago, CO	41 5	594	-5	-3	1	M	94	91	88	15	78	76	75
Danville	40 1	558	-6	-1	4	M	96	94	91	21	79	78	76
Decatur	39 5	670	-6	0	4	M	96	93	91	21	78	77	77
Dixon	41 5	696	-13	-7	-3	M	93	91	89	23	78	77	75
Elgin	42 0	820	-14	-8	-4	M	92	90	87	21	78	76	75
Freeport	42 2	780	-16	-10	-6	M	92	90	87	24	78	77	75
Galesburg	41 0	771	-10	-4	0	M	95	92	89	22	79	78	76
Greenville	39 0	563	-3	3	7	M	96	94	92	21	79	78	77
Joliet AP	41 3	588	-11	-5	-1	M	94	92	89	20	78	77	75
Kankakee	41 1	625	-10	-4	1	M	94	92	89	21	78	77	76
La Salle/Peru	41 2	520	-9	-3	1	M	94	93	90	22	78	77	76
Macomb	40 3	702	-5	-3	1	M	95	93	90	22	79	78	77
Moline AP	41 3	582	-12	-7	-3	M	94	91	88	23	79	77	76
Mt. Vernon	38 2	500	0	6	10	M	97	95	92	21	79	78	77
Peoria AP	40 4	652	-8	-2	2	M	94	92	89	22	78	77	76
Quincy AP	40 0	762	-8	-2	2	M	97	95	92	22	80	79	77
Rantoul, Chanute AFB	40 2	740	-7	-1	3	M	94	92	89	21	78	77	76
Rockford	42 1	724	-13	-7	-3	M	92	90	87	24	77	76	75
Springfield AP	39 5	587	-7	-1	4	M	95	92	90	21	79	78	77
Waukegan	42 2	680	-11	-5	-1	M	92	90	87	21	77	76	75
INDIANA													
Anderson	40 0	847	-5	0	5	M	93	91	88	22	78	77	76
Bedford	38 5	670	-3	3	7	M	95	93	90	22	79	78	77
Bloomington	39 1	520	-3	3	7	M	95	93	90	22	79	78	76
Columbus, Bakalar AFB	39 2	661	-3	3	7	M	95	92	90	22	79	78	76
Crawfordsville	40 0	752	-3	-2	2	M	95	93	90	22	79	77	76
Evansville AP	38 0	381	1	6	10	M	96	94	91	22	79	78	77
Fort Wayne AP	41 0	791	-5	0	5	M	93	91	88	24	77	76	75
Goshen AP	41 3	823	-10	-4	0	M	92	90	87	23	77	76	74
Hobart	41 3	600	-10	-4	0	M	93	91	88	21	78	76	75
Huntington	40 4	802	-8	-2	2	M	94	92	89	23	78	76	75
Indianapolis AP	39 4	793	-5	0	4	M	93	91	88	22	78	77	76
Jeffersonville	38 2	455	-6	0	13	M	96	94	91	23	78	77	75
Kokomo	40 3	790	-6	0	4	M	94	92	89	22	78	76	75
Lafayette	40 2	600	-7	-1	3	M	94	92	89	22	78	77	76

Climatic Conditions for United States and Canada (Continued)*.a

Col. 1 State and Station ^b	Col. 2 Latitude ^c	Col. 3 Elev. ^d Ft	Winter				Summer						
			Median of Annual Ex- tremes	Col. 4		Col. 5 Coincident Wind Velocity ^e	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range ^f	Col. 8 Design Wet-Bulb		
				99%	97½%		1%	2½%	5%		1%	2½%	5%
INDIANA (continued)													
La Porte	41 3	810	-10	-4	0	M	93	91	88	22	77	76	74
Marion	40 3	791	-8	-2	2	M	93	91	88	23	78	76	75
Muncie	40 1	955	-8	-2	2	M	93	91	88	22	78	77	75
Peru, Bunker Hill AFB	40 4	804	-9	-3	1	M	91	89	86	22	77	76	74
Richmond AP	39 5	1138	-7	-1	3	M	93	91	88	22	78	77	75
Shelbyville	39 3	765	-4	2	6	M	94	92	89	22	78	77	76
South Bend AP	41 4	773	-6	-2	3	M	92	89	87	22	77	76	74
Terre Haute AP	39 3	601	-3	3	7	M	95	93	91	22	79	78	77
Valparaiso	41 2	801	-12	-6	-2	M	92	90	87	22	78	76	75
Vincennes	38 4	420	-1	5	9	M	96	94	91	22	79	78	77
IOWA													
Ames	42 0	1004	-17	-11	-7	M	94	92	89	23	79	78	76
Burlington AP	40 5	694	-10	-4	0	M	95	92	89	22	80	78	77
Cedar Rapids AP	41 5	863	-14	-8	-4	M	92	90	87	23	78	76	75
Clinton	41 5	595	-13	-7	-3	M	92	90	87	23	78	77	76
Council Bluffs	41 2	1210	-14	-7	-3	M	97	94	91	22	79	77	76
Des Moines AP	41 3	948r	-13	-7	-3	M	95	92	89	23	79	78	76
Dubuque	42 2	1065	-17	-11	-7	M	92	90	87	22	78	76	75
Fort Dodge	42 3	1111	-18	-12	-8	M	94	92	89	23	78	77	75
Iowa City	41 4	645	-14	-8	-4	M	94	91	88	22	79	77	76
Keokuk	40 2	526	-9	-3	1	M	95	93	90	22	79	78	77
Marshalltown	42 0	898	-16	-10	-6	M	93	91	88	23	79	77	76
Mason City AP	43 1	1194	-20	-13	-9	M	91	88	85	24	77	75	74
Newton	41 4	946	-15	-9	-5	M	95	93	90	23	79	77	76
Ottumwa AP	41 1	842	-12	-6	-2	M	95	93	90	22	79	78	76
Sioux City AP	42 2	1095	-17	-10	-6	M	96	93	90	24	79	77	76
Waterloo	42 3	868	-18	-12	-8	M	91	89	86	23	78	76	75
KANSAS													
Atchison	39 3	945	-9	-2	2	M	97	95	92	23	79	78	77
Chanute AP	37 4	977	-3	3	7	H	99	97	95	23	79	78	77
Dodge City AP	37 5	2594	-5	3	7	M	99	97	95	25	74	73	72
El Dorado	37 5	1282	-3	4	8	H	101	99	96	24	78	77	76
Emporia	38 2	1209	-4	3	7	H	99	97	94	25	78	77	76
Garden City AP	38 0	2882	-10	-1	3	M	100	98	96	28	74	73	72
Goodland AP	39 2	3645	-10	-2	4	M	99	96	93	31	71	70	69
Great Bend	38 2	1940	-5	2	6	M	101	99	96	28	77	76	75
Hutchinson AP	38 0	1524	-5	2	6	H	101	99	96	28	77	76	75
Liberal	37 0	2838	-4	4	8	M	102	100	99	28	74	73	71
Manhattan, Fort Riley	39 0	1076	-7	-1	4	H	101	98	95	24	79	78	77
Parsons	37 2	908	-2	5	9	H	99	97	94	23	79	78	77
Russell AP	37 0	1864	-7	0	4	M	102	100	97	29	78	76	75
Salina	38 5	1271	-4	3	7	H	101	99	96	26	78	76	75
Topeka AP	39 0	877	-4	3	6	M	99	96	94	24	79	78	77
Wichita AP	37 4	1321	-1	5	9	H	102	99	96	23	77	76	75
KENTUCKY													
Ashland	38 3	551	1	6	10	L	94	92	89	22	77	76	75
Bowling Green AP	37 0	535	1	7	11	L	97	95	93	21	79	78	77
Corbin AP	37 0	1175	0	5	9	L	93	91	89	23	79	77	76
Covington AP	39 0	869	-3	3	8	L	93	90	88	22	77	76	75
Hopkinsville, Campbell AFB	36 4	540	-4	10	14	L	97	95	92	21	79	78	77
Lexington AP	38 0	979	0	6	10	M	94	92	90	22	78	77	76
Louisville AP	38 1	474	1	8	12	L	96	93	91	23	79	78	77
Madisonville	37 2	439	1	7	11	L	96	94	92	22	79	78	77
Owensboro	37												

Climatic Conditions for United States and Canada (Continued)*

Col. 1 State and Station	Col. 2 Latitude	Col. 3 Elev. Ft.	Winter			Summer								
			Col. 4		Col. 5 Coincident Wind Ve- locity	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range	Col. 8 Design Wet-Bulb				
			Median of Annual Ex- tremes	99%		97½%	1%	2½%		5%	1%	2½%	5%	
														99%
LOUISIANA (continued)														
Monroe AP	32 3	78	18	23	27	L	98	96	95	20	81	81	80	
Natchitoches	31 5	120	17	22	26	M	99	97	96	20	81	80	79	
New Orleans AP	30 0	3	29	32	35	M	93	91	90	16	81	80	79	
Shreveport AP	32 3	252	18	22	26	M	99	96	94	20	81	80	79	
MAINE														
Augusta AP	44 2	350	-13	-7	-3	M	88	86	83	22	74	73	71	
Bangor, Dow AFB	44 5	162	-14	-8	-4	M	88	85	81	22	75	73	71	
Caribou AP	46 5	624	-24	-18	-14	L	85	81	78	21	72	70	68	
Lewiston	44 0	182	-14	-8	-4	M	88	86	83	22	74	73	71	
Millinocket AP	45 4	405	-22	-16	-12	L	87	85	82	22	74	72	70	
Portland AP	43 4	81	-14	-5	0	L	88	85	81	22	75	73	71	
Waterville	44 3	69	-15	-9	-5	M	88	86	82	22	74	73	71	
MARYLAND														
Baltimore AP	39 1	146	8	12	15	M	94	91	89	21	79	78	77	
Baltimore CO	39 2	14	12	16	20	M	94	92	89	17	79	78	77	
Cumberland	39 4	945	0	5	9	L	94	92	89	22	76	75	74	
Frederick AP	39 2	294	2	7	11	M	94	92	89	22	78	77	76	
Hagerstown	39 4	660	1	6	10	L	94	92	89	22	77	76	75	
Salisbury	38 2	52	10	14	18	M	92	90	87	18	79	78	77	
MASSACHUSETTS														
Boston AP	42 2	15	-1	-6	10	H	91	88	85	16	76	74	73	
Clinton	42 2	398	-8	-2	2	M	87	85	82	17	75	74	72	
Fall River	41 4	190	-1	-5	9	H	88	86	83	18	75	74	73	
Framingham	42 2	170	-7	-1	3	M	91	89	86	17	76	74	73	
Gloucester	42 3	10	-4	-2	6	H	86	84	81	15	74	73	72	
Greenfield	42 3	205	-12	-6	-2	M	89	87	84	23	75	74	73	
Lawrence	42 4	57	-9	-3	1	M	90	88	85	21	76	74	72	
Lowell	42 3	90	-7	-1	3	M	91	89	86	21	76	74	72	
New Bedford	41 4	70	3	9	13	H	86	84	81	19	75	73	72	
Pittsfield AP	42 3	1170	-11	-5	-1	M	86	84	81	23	74	72	71	
Springfield, Westover AFB	42 1	247	-8	-3	2	M	91	88	85	19	76	74	73	
Taunton	41 5	20	-9	-4	0	H	88	86	83	18	76	75	74	
Worcester AP	42 2	986	-8	-3	1	M	89	87	84	18	75	73	71	
MICHIGAN														
Adrian	41 5	754	-6	0	4	M	93	91	88	23	76	75	74	
Alpena AP	45 0	689	-11	-5	-1	M	87	85	82	27	74	73	71	
Battle Creek AP	42 2	939	-6	1	5	M	92	89	86	23	76	74	73	
Benton Harbor AP	42 1	649	-7	-1	3	M	90	88	85	20	76	74	73	
Detroit Met. CAP	42 2	633	0	4	8	M	92	88	85	20	76	75	74	
Escanaba	45 4	594	-13	-7	-3	M	82	80	77	17	73	71	69	
Flint AP	43 0	766	-7	-1	3	M	89	87	84	25	76	75	74	
Grand Rapids AP	42 5	691	-3	2	6	M	91	89	86	24	76	74	73	
Holland	42 5	612	-4	2	6	M	90	88	85	22	76	74	73	
Jackson AP	42 2	1003	-4	0	4	M	92	89	86	23	76	75	74	
Kalamazoo	42 1	930	-5	1	5	M	92	89	86	23	76	75	74	
Lansing AP	42 5	852	-4	2	6	M	89	87	84	24	76	75	73	
Marquette CO	46 3	677	-14	-8	-4	L	88	86	83	18	73	71	69	
Mt. Pleasant	43 4	796	-9	-3	1	M	89	87	84	24	75	74	73	
Muskegon AP	43 1	627	-2	4	8	M	87	85	82	21	75	74	73	
Pontiac	42 4	974	-6	0	4	M	90	88	85	21	76	75	73	
Port Huron	43 0	586	-6	-1	3	M	90	88	85	21	76	74	73	
Saginaw AP	43 3	662	-7	-1	3	M	88	86	83	23	76	75	73	
Sault Ste. Marie AP	46 3	721	-18	-12	-8	L	83	81	78	23	73	71	69	
Traverse City AP	44 4	618	-6	0	4	M	89	86	83	22	75	73	72	
Ypsilanti	42 1	777	-3	-1	5	M	92	89	86	22	76	74	73	
MINNESOTA														
Albert Lea	43 4	1235	-20	-14	-10	M	91	89	86	24	77	76	74	
Alexandria AP	45 5	1421	-26	-19	-15	L	90	88	85	24	76	74	72	
Bemidji AP	47 3	1392	-38	-32	-28	L	87	84	81	24	73	72	71	
Brainerd	46 2	1214	-31	-24	-20	L	88	85	82	24	74	73	72	
Duluth AP	46 5	1426	-25	-19	-15	M	85	82	79	22	73	71	69	
Faribault	44 2	1190	-23	-16	-12	L	90	88	85	24	77	75	74	
Fergus Falls	46 1	1210	-28	-21	-17	L	92	89	86	24	75	74	72	
International Falls AP	48 3	1179	-35	-29	-24	L	86	82	79	26	72	69	68	

Climatic Conditions for United States and Canada (Continued)*

Col. 1 State and Station	Col. 2 Latitude	Col. 3 Elev. Ft.	Winter			Summer								
			Col. 4		Col. 5 Coincident Wind Ve- locity	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range	Col. 8 Design Wet-Bulb				
			Median of Annual Ex- tremes	99%		97½%	1%	2½%		5%	1%	2½%	5%	
														99%
MINNESOTA (continued)														
Mankato	44 1	785	-23	-16	-12	L	91	89	86	24	77	75	74	
Minneapolis/St. Paul AP	44 5	822	-19	-14	-10	L	92	89	86	22	77	75	74	
Rochester AP	44 0	1297	-23	-17	-13	M	90	88	85	24	77	75	74	
St. Cloud AP	45 4	1084	-26	-20	-16	L	90	88	85	24	77	75	73	
Virginia	47 3	1435	-32	-25	-21	L	86	83	80	23	73	71	69	
Willmar	45 1	1133	-25	-18	-14	L	91	88	85	24	77	75	73	
Winona	44 1	652	-19	-12	-8	M	91	89	86	24	77	76	74	
MISSISSIPPI														
Biloxi, Keesler AFB	30 2	25	26	30	32	M	93	92	90	16	82	81	80	
Clarksdale	34 1	178	14	20	24	L	98	96	95	21	81	80	79	
Columbus AFB	33 4	224	13	18	22	L	97	95	93	22	79	79	78	
Greenville AFB	33 3	139	16	21	24	L	98	96	94	21	81	80	79	
Greenwood	33 3	128	14	19	23	L	98	96	94	21	81	80	79	
Hattiesburg	31 2	200	15	22	26	L	97	95	94	21	80	79	78	
Jackson AP	32 2	330	17	21	24	L	98	96	94	21	79	78	78	
Laurel	31 4	264	18	22	26	L	97	95	94	21	80	79	78	
McComb AP	31 2	458	18	22	26	L	96	94	93	18	80	79	79	
Meridian AP	32 2	294	15	20	24	L	97	95	94	22	80	79	78	
Natchez	31 4	168	18	22	26	L	96	94	93	21	80	80	79	
Tupelo	34 2	289	13	18	22	L	98	96	95	22	80	79	78	
Vicksburg CO	32 2	234	18	23	26	L	97	95	94	21	80	80	79	
MISSOURI														
Cape Girardeau	37 1	330	2	8	12	M	98	96	94	21	80	79	78	
Columbia AP	39 0	778	-4	2	6	M	97	95	92	22	79	78	77	
Farmington AP	37 5	928	-2	4	8	M	97	95	93	22	79	78	77	
Hannibal	39 4	489	-7	-1	4	M	96	94	91	22	79	78	77	
Jefferson City	38 4	640	-4	2	6	M	97	95	93	23	79	78	77	
Joplin AP	37 1	982	1	7	11	M	97	95	93	24	79	78	77	
Kansas City AP	39 1	742	-2	4	8	M	100	97	94	20	79	77	76	
Kirkville AP	40 1	966	-13	-7	-3	M	96	94	91	24	79	78	77	
Mexico	39 1	775	-7	-1	3	M	96	94	91	22	79	78	77	
Moberly	39 3	850	-8	-2	2	M	96	94	91	23	79	78	77	
Poplar Bluff	36 5	322	3	9	13	M	98	96	94	22	80	79	78	
Rolla	38 0	1202	-3	3	7	M	97	95	93	22	79	78	77	
St. Joseph AP	39 5	809	-8	-1	3	M	97	95	92	23	79	78	77	
St. Louis AP	38 5	535	-2	4	8	M	98	95	92	21	79	78	77	
St. Louis CO	38 4	465	1	7	11	M	96	94	92	18	79	78	77	
Sedalia, Whiteman AFB	38 4	838	-2	4	9	M	97	94	92	22	79	77	76	
Sikeston	36 5	318	4	10	14	L	98	96	94	21	80	79	78	
Springfield AP	37 1	1265	0	5	10	M	97	94	91	23	78	77	76	
MONTANA														
Billings AP	45 5	3567	-19	-10	-6	L	94	91	88	31	68	66	65	
Bozeman	45 5	4556	-25	-15	-11	L	88	85	82	32	61	60	59	
Butte AP	46 0	5526r	-34	-24	-16	VL	86	83	80	35	60	59	57	
Cut Bank AP	48 4	3238r	-32	-23	-17	L	89	86	82	35	65	63	61	
Glasgow AP	48 1	2277	-33	-25	-20	L	96	93	89	29	69	67	65	
Glendive	47 1	2076	-28	-20	-16	L	96	93	90	29	71	69	68	
Great Falls AP	47 3	3664r	-29	-20	-16	L	91	88	85	28	64	63	61	
Havre	48 3	2488	-32	-22	-15	M	91	87	84	33	66	64	63	
Helena AP	46 4	3893	-27	-17	-13	L	90	87	84	32	65	63	61	
Kalispell AP	48 2	2965	-17	-7	-3	VL	88	84	81	34	65	63	62	
Lewiston AP	47 0	4132	-27	-18	-14	L	89	86	83	30	65	63	62	
Livingston AP	45 4	4653	-26	-17	-13	L	91	88	85	32	63	62		

Climatic Conditions for United States and Canada (Continued)*-a

Col. 1 State and Station	Col. 2 Latitude	Col. 3 Elev. Ft	Winter			Summer							
			Col. 4		Col. 5 Coincidence Wind Ve- locity	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range	Col. 8 Design Wet-Bulb			
			Median of Annual Ex- tremes	99%		97½%	1%	2½%		5%	1%	2½%	5%
NEBRASKA (continued)													
McCook	40 1	2565	-12	-4	0	M	99	97	94	28	74	72	71
Norfolk	42 0	1532	-18	-11	-7	M	97	95	92	30	78	76	75
North Platte AP	41 1	2779	-13	-6	-2	M	97	94	90	28	74	73	72
Omaha AP	41 2	978	-12	-5	-1	M	97	94	91	22	79	78	76
Scottsbluff AP	41 5	3950	-16	-8	-4	M	96	94	91	31	70	69	67
Sidney AP	41 1	4292	-15	-7	-2	M	95	92	89	31	70	69	67
NEVADA													
Carson City	39 1	4675	-4	3	7	VL	93	91	88	42	62	61	60
Elko AP	40 5	5075	-21	-13	-7	VL	94	92	90	42	64	62	61
Ely AP	39 1	6257	-15	-6	-2	VL	90	88	86	39	60	59	58
Las Vegas AP	36 1	2162	18	23	26	VL	108	106	104	30	72	71	70
Lovelock AP	40 0	3900	0	7	11	VL	98	96	93	42	65	64	62
Reno AP	39 3	4404	-2	2	7	VL	95	92	90	45	64	62	61
Reno CO	39 3	4490	8	12	17	VL	94	92	89	45	64	62	61
Tonopah AP	38 0	5426	2	9	13	VL	95	92	90	40	64	63	62
Winnemucca AP	40 5	4299	-8	1	5	VL	97	95	93	42	64	62	61
NEW HAMPSHIRE													
Berlin	44 3	1110	-25	-19	-15	L	87	85	82	22	73	71	70
Claremont	43 2	420	-19	-13	-9	L	89	87	84	24	74	73	72
Concord AP	43 1	339	-17	-11	-7	M	91	88	85	26	75	73	72
Keene	43 0	490	-17	-12	-8	M	90	88	85	24	75	73	72
Laconia	43 3	505	-22	-16	-12	M	89	87	84	25	74	73	72
Manchester, Greater AFB	43 0	253	-11	-5	1	M	92	89	86	24	76	74	73
Portsmouth, Pease AFB	43 1	127	-8	-2	3	M	88	86	83	22	75	73	72
NEW JERSEY													
Atlantic City CO	39 3	11	10	14	18	H	91	88	85	18	78	77	76
Long Branch	40 2	20	4	9	13	H	93	91	88	18	77	76	75
Newark AP	40 4	11	6	11	15	M	94	91	88	20	77	76	75
New Brunswick	40 3	86	3	8	12	M	91	89	86	19	77	76	75
Paterson	40 5	100	3	8	12	M	93	91	88	21	77	76	75
Phillipsburg	40 4	180	1	6	10	L	93	91	88	21	77	76	75
Trenton CO	40 1	144	7	12	16	M	92	90	87	19	78	77	76
Vineland	39 3	95	7	12	16	M	93	90	87	19	78	77	76
NEW MEXICO													
Alamogordo, Holloman AFB	32 5	4070	12	18	22	L	100	98	96	30	70	69	68
Albuquerque AP	35 0	5310	6	14	17	L	96	94	92	27	66	65	64
Artesia	32 5	3375	9	16	19	L	101	99	97	30	71	70	69
Carlsbad AP	32 2	3234	11	17	21	L	102	100	97	30	72	71	70
Clovis AP	34 3	4279	-2	14	17	L	99	97	95	28	70	69	68
Farmington AP	36 5	5495	-3	6	9	VL	95	93	91	30	66	65	64
Gallup	35 3	6465	-13	-5	-1	VL	92	90	87	32	64	63	62
Grants	35 1	6520	-15	-7	-3	VL	91	89	86	32	64	63	62
Hobbs AP	32 4	3664	9	15	19	L	101	99	96	29	72	71	70
Las Cruces	32 2	3900	13	19	23	L	102	100	97	30	70	69	68
Los Alamos	35 5	7410	-4	5	9	L	88	86	83	32	64	63	62
Raton AP	36 5	6379	-11	-2	2	L	92	90	88	34	66	65	64
Roswell, Walker AFB	33 2	3643	5	16	19	L	101	99	97	33	71	70	69
Santa Fe CO	35 4	7045	-2	7	11	L	90	88	85	28	65	63	62
Silver City AP	32 4	5373	8	14	18	VL	95	93	91	30	68	67	66
Socorro AP	34 0	4617	6	13	17	L	99	97	94	30	67	66	65
Tucumcari AP	35 1	4053	1	9	13	L	99	97	95	28	71	70	69
NEW YORK													
Albany AP	42 5	277	-14	-5	0	L	91	88	85	23	76	74	73
Albany CO	42 5	19	-5	1	5	L	91	89	86	20	76	74	73
Auburn	43 0	715	-10	-2	2	M	89	87	84	22	75	73	72
Batavia	43 0	900	-7	-1	3	M	89	87	84	22	75	74	72
Binghamton CO	42 1	858	-8	-2	2	L	91	89	86	20	74	72	71
Buffalo AP	43 0	705r	-3	3	6	M	88	86	83	21	75	73	72
Cortland	42 4	1129	-11	-5	-1	L	90	88	85	23	75	73	72
Dunkirk	42 3	590	-2	4	8	M	88	86	83	18	75	74	72
Elmira AP	42 1	860	-5	1	5	L	92	90	87	24	75	73	72
Geneva	42 5	590	-8	-2	2	M	91	89	86	22	75	73	72
Glens Falls	43 2	321	-17	-11	-7	L	85	86	83	23	74	72	71
Gloversville	43 1	770	-12	-6	-2	L	89	87	84	23	75	73	71
Hornell	42 2	1325	-15	-9	-5	L	87	85	82	24	74	72	71

Climatic Conditions for United States and Canada (Continued)*-a

Col. 1 State and Station	Col. 2 Latitude	Col. 3 Elev. Ft	Winter			Summer							
			Col. 4		Col. 5 Coincidence Wind Ve- locity	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range	Col. 8 Design Wet-Bulb			
			Median of Annual Ex- tremes	99%		97½%	1%	2½%		5%	1%	2½%	5%
NEW YORK (continued)													
Ithaca	42 3	950	-10	-4	0	L	91	88	85	24	75	73	72
Jamestown	42 1	1390	-5	1	5	M	88	86	83	20	75	73	72
Kingston	42 0	279	-8	-2	2	L	92	90	87	22	76	74	73
Lockport	43 1	520	-4	2	6	M	87	85	82	21	75	74	72
Massena AP	45 0	202r	-22	-16	-12	M	86	84	81	20	75	74	72
Newburgh-Stewart AFB	41 3	460	-4	2	6	M	92	89	86	21	78	76	74
NYC-Central Park	40 5	132	6	11	15	H	94	91	88	17	77	76	75
NYC-Kennedy AP	40 4	16	12	17	21	H	91	87	84	16	77	76	75
NYC-LaGuardia AP	40 5	19	-7	12	16	H	93	90	87	16	77	76	75
Niagara Falls AP	43 1	596	-2	4	7	M	88	86	83	20	75	74	73
Olean	42 1	1420	-13	-8	-3	L	87	85	82	23	74	72	71
Oneonta	42 3	1150	-13	-7	-3	L	89	87	84	24	74	72	71
Oneco CO	43 3	300	-4	2	6	M	86	84	81	20	75	74	72
Plattsburg AFB	44 4	165	-16	-10	-6	L	86	84	81	22	74	73	71
Poughkeepsie	41 4	103	-6	-1	3	L	93	90	87	21	77	75	74
Rochester AP	43 1	543	-5	2	5	M	91	88	85	22	75	74	72
Rome-Griffiss AFB	43 1	515	-13	-7	-3	L	90	87	84	22	76	74	73
Schenectady	42 5	217	-11	-5	-1	L	90	88	85	22	75	73	72
Suffolk County AFB	40 5	57	4	9	13	H	87	84	81	16	76	75	74
Syracuse AP	43 1	424	-10	-2	2	M	90	87	85	20	76	74	73
Utica	43 1	714	-12	-6	-2	L	89	87	84	22	75	73	72
Watertown	44 0	497	-20	-14	-10	M	86	84	81	20	75	74	72
NORTH CAROLINA													
Asheville AP	35 3	2170r	8	13	17	L	91	88	86	21	75	74	73
Charlotte AP	35 1	735	13	18	22	L	96	94	92	20	78	77	76
Durham	36 0	406	11	15	19	L	94	92	89	20	78	77	76
Elizabeth City AP	36 2	10	14	18	22	M	93	91	89	18	80	79	78
Fayetteville, Pope AFB	35 1	95	13	17	20	L	97	94	92	20	80	79	78
Goldboro, Seymour-Johnson AFB	35 2	88	14	18	21	M	95	92	90	18	80	79	78
Greensboro AP	36 1	897	9	14	17	L	94	91	89	21	77	76	75
Greenville	35 4	25	14	18	22	M	95	93	90	19	81	80	79
Henderson	36 2	510	8	12	16	L	94	92	89	20	79	78	77
Hickory	35 4	1165	9	14	18	L	93	91	88	21	77	76	75
Jacksonville	34 5	24	17	21	25	M	94	92	89	18	81	80	79
Lumberton	34 4	132	14	18	22	L	95	93	90	20	81	80	79
New Bern AP	35 1	17	14	18	22	L	94	92	89	18	81	80	79
Raleigh/Durham AP	35 5	433	13	16	20	L	95	92	90	20	79	78	77
Rocky Mount	36 0	81	12	16	20	L	95	93	90	19	80	79	78
Wilmington AP	34 2	30	19	23	27	L	93	91	89	18	82	81	80
Winston-Salem AP	36 1	967	9	14	17	L	94	91	89	20	77	76	75
NORTH DAKOTA													
Bismarck AP	46 5	1647	-31	-24	-19	VL	95	91	88	27	74	72	70
Devil's Lake	48 1	1471	-30	-23	-19	M	93	89	86	25	73	71	69
Dickinson AP	46 5	2595	-31	-23	-19	L	96	93	90	25	72	70	68
Fargo AP	46 5	900	-28	-22	-17	L	92	88	85	25	76	74	72
Grand Forks AP	48 0	832	-30	-26	-23	L	91	87	84	25	74	72	70
Jamestown AP	47 0	1492	-29	-22	-18	L	95	91	88	26	75	73	71
Minot AP	48 2	1713	-31	-24	-20	M	91	88	84	25	72	70	68
Williston	48 1	1877	-28	-21	-17	M	94	90	87	25	71	69	67
OHIO													
Akron/Canton AP	41 0	1210	-5	1	6	M	89	87	84	21	75	73	72
Ashtabula													

Climatic Conditions for United States and Canada (Continued)*-*

Col. 1 State and Station ^b	Col. 2 Latitude ^a	Col. 3 Elev. ^d Ft.	Winter			Summer							
			Median of Annual Ex- tremes	Col. 4		Col. 5 Coinci- dent Wind Ve- locity ^e	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range ^f	Col. 8 Design Wet-Bulb		
				99%	97½%		1%	2½%	5%		1%	2½%	5%
OHIO (continued)													
Fremont	41 2	600	-7	-1	3	M	92	90	87	24	76	75	74
Hamilton	39 2	650	-2	4	8	M	94	92	90	22	78	77	76
Lancaster	39 4	920	-5	1	5	M	93	91	88	23	77	76	75
Lima	40 4	860	-6	0	4	M	93	91	88	24	77	76	75
Mansfield AP	40 5	1297	-7	1	3	M	91	89	86	22	76	75	74
Marion	40 4	920	-5	1	6	M	93	91	88	23	77	76	75
Middletown	39 3	635	-3	3	7	M	93	91	88	22	77	76	75
Newark	40 1	825	-7	-1	3	M	92	90	87	23	77	76	75
Norwalk	41 1	720	-7	-1	3	M	92	90	87	22	76	75	74
Portsmouth	38 5	530	-0	5	9	L	94	92	89	22	77	76	75
Sandusky CO	41 3	606	-2	4	8	M	92	90	87	21	76	75	74
Springfield	40 0	1020	-3	3	7	M	93	90	88	21	77	76	75
Stuebenville	40 2	992	-2	4	9	M	91	89	86	22	76	75	74
Toledo AP	41 4	676r	-5	1	5	M	92	90	87	25	77	75	74
Warren	41 2	900	-6	0	4	M	90	88	85	23	75	74	73
Wooster	40 5	1030	-7	-1	3	M	90	88	85	22	76	75	74
Youngstown AP	41 2	1178	-5	1	6	M	89	86	84	23	75	74	73
Zanesville AP	40 0	881	-7	-1	3	M	92	89	87	23	77	76	75
OKLAHOMA													
Ada	34 5	1015	6	12	16	H	102	100	98	23	79	78	77
Altus AFB	34 4	1390	7	14	18	H	103	101	99	25	77	76	75
Ardmore	34 2	880	9	15	19	H	103	101	99	23	79	78	77
Barlesville	36 5	715	-1	5	9	H	101	99	97	23	79	78	77
Chickasha	35 0	1085	5	12	16	H	103	101	99	24	77	76	75
Enid-Vance AFB	36 2	1287	3	10	14	H	103	100	98	24	78	77	76
Lawton AP	34 3	1108	6	13	16	H	103	101	98	24	78	77	76
McAlester	34 5	760	7	13	17	H	102	100	98	23	79	78	77
Muskogee AP	35 4	610	6	12	16	M	102	99	96	23	79	78	77
Norman	35 1	1109	5	11	15	H	101	99	97	24	78	77	76
Oklahoma City AP	35 2	1280	4	11	15	H	100	97	95	23	78	77	76
Ponca City	36 4	996	1	8	12	H	102	100	97	24	78	77	76
Seminole	35 2	865	6	12	16	H	102	100	98	23	78	77	76
Stillwater	36 1	884	2	9	13	H	101	99	97	24	78	77	76
Tulsa AP	36 1	650	4	12	16	H	102	99	96	22	79	78	77
Woodward	36 3	1900	-3	4	8	H	103	101	98	26	76	74	73
OREGON													
Albany	44 4	224	17	23	27	VL	91	88	84	31	69	67	65
Astoria AP	46 1	8	-22	27	30	M	79	76	72	16	61	60	59
Baker AP	44 5	3368	-10	-3	1	VL	94	92	89	30	66	65	63
Bend	44 0	3599	-7	0	4	VL	89	87	84	33	64	62	61
Corvallis	44 3	221	17	23	27	VL	91	88	84	31	69	67	65
Eugene AP	44 1	364	16	22	26	VL	91	88	84	31	69	67	65
Grants Pass	42 3	925	16	22	26	VL	94	92	89	33	68	66	65
Klamath Falls AP	42 1	4091	-5	1	5	VL	89	87	84	36	63	62	61
Medford AP	42 2	1298	15	21	23	VL	98	94	91	35	70	68	66
Pendleton AP	45 4	1492	-2	3	10	VL	97	94	91	29	66	65	63
Portland AP	45 4	21	17	21	24	L	89	85	81	23	69	67	66
Portland CO	45 3	57	21	26	29	L	91	88	84	21	69	68	67
Roseburg AP	43 1	505	19	25	29	VL	93	91	88	30	69	67	65
Salem AP	45 0	195	15	21	25	VL	92	88	84	31	69	67	66
The Dalles	45 4	102	7	13	17	VL	93	91	88	28	70	68	67
PENNSYLVANIA													
Allentown AP	40 4	376	-2	3	5	M	92	90	87	22	77	75	74
Altoona CO	40 2	1468	-4	1	5	L	89	87	84	23	74	73	72
Butler	40 4	1100	-8	-2	2	L	91	89	86	22	75	74	73
Chambersburg	40 0	640	0	5	9	L	94	92	89	23	76	75	74
Erie AP	42 1	732	1	7	11	M	88	85	82	18	76	74	73
Harrisburg AP	40 1	335	-4	9	13	L	92	89	86	21	76	75	74
Johnstown	40 2	1214	-4	1	5	L	91	87	85	23	74	73	72
Lancaster	40 1	255	-3	2	6	L	92	90	87	22	77	76	75
Meadville	41 4	1065	-6	0	4	M	88	86	83	21	75	73	72
New Castle	41 0	825	-1	1	4	M	91	89	86	23	75	74	73

Climatic Conditions for United States and Canada (Continued)*-*

Col. 1 State and Station ^b	Col. 2 Latitude ^a	Col. 3 Elev. ^d Ft.	Winter			Summer							
			Median of Annual Ex- tremes	Col. 4		Col. 5 Coinci- dent Wind Ve- locity ^e	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range ^f	Col. 8 Design Wet-Bulb		
				99%	97½%		1%	2½%	5%		1%	2½%	5%
PENNSYLVANIA (continued)													
Philadelphia AP	39 5	7	-7	11	15	M	93	90	87	21	78	77	76
Pittsburgh AP	40 3	1137	-1	5	9	M	90	87	85	22	75	74	73
Pittsburgh CO	40 3	749r	-1	7	11	M	90	88	85	19	75	74	73
Reading CO	40 2	226	1	6	9	M	92	90	87	19	77	76	75
Scranton/Wilkes-Barre	41 2	940	-3	2	6	L	89	87	84	19	75	74	73
State College	40 5	1175	-3	2	6	L	89	87	84	23	74	73	72
Sunbury	40 5	480	-2	3	7	L	91	89	86	22	76	75	74
Uniontown	39 5	1040	-1	4	8	L	90	88	85	22	75	74	73
Warren	41 5	1280	-8	-3	1	L	89	87	84	24	75	73	72
West Chester	40 0	440	4	9	13	M	92	90	87	20	77	76	75
Williamsport AP	41 1	527	-5	1	5	L	91	89	86	23	76	75	74
York	40 1	390	-1	4	8	L	93	91	88	22	77	76	75
RHODE ISLAND													
Newport	41 3	20	1	5	11	H	86	84	81	16	75	74	73
Providence AP	41 4	55	0	6	10	M	89	86	83	19	76	75	74
SOUTH CAROLINA													
Anderson	34 3	764	13	18	22	L	96	94	91	21	77	76	75
Charleston AFB	32 5	41	19	23	27	L	94	92	90	18	81	80	79
Charleston CO	32 5	9	23	26	30	L	95	93	90	13	81	80	79
Columbia AP	34 0	217	16	20	23	L	98	96	94	22	79	79	78
Florence AP	34 1	146	16	21	25	L	96	94	92	21	80	79	78
Georgetown	33 2	14	19	23	26	L	93	91	88	18	81	80	79
Greenville AP	34 5	657	14	19	23	L	95	93	91	21	77	76	75
Greenwood	34 1	671	15	19	23	L	97	95	92	21	78	77	76
Orangeburg	33 3	244	17	21	25	L	97	95	92	20	80	79	78
Rock Hill	35 0	470	13	17	21	L	97	95	92	20	78	77	76
Spartanburg AP	35 0	816	13	18	22	L	95	93	90	20	77	76	75
Sumter-Shaw AFB	34 0	291	18	23	26	L	96	94	92	21	80	79	78
SOUTH DAKOTA													
Aberdeen AP	45 3	1296	-29	-22	-18	L	95	92	89	27	77	75	74
Brookings	44 2	1642	-26	-19	-15	M	93	90	87	25	77	75	74
Huron AP	44 3	1282	-24	-16	-12	L	97	93	90	28	77	75	74
Mitchell	43 5	1346	-22	-15	-11	M	96	94	91	28	77	76	74
Pierre AP	44 2	1718r	-21	-13	-9	M	98	96	93	29	76	74	73
Rapid City AP	44 0	3165	-17	-9	-6	M	96	94	91	28	72	71	69
Sioux Falls AP	43 4	1420	-21	-14	-10	M	95	92	89	24	77	75	74
Watertown AP	45 0	1746	-27	-20	-16	L	93	90	87	26	76	74	73
Yankton	43 0	1280	-18	-11	-7	M	96	94	91	25	78	76	75
TENNESSEE													
Athens	33 3	940	10	14	18	L	96	94	91	22	77	76	75
Bristol-Tri City AP	36 3	1519	-1	11	16	L	92	90	88	22	76	75	74
Chattanooga AP	35 0	670	11	15	19	L	97	94	92	22	78	78	77
Clarksville	36 4	470	6	12	16	L	98	96	94	21	79	78	77
Columbia	35 4	690	8	13	17	L	97	95	93	21	79	78	77
Dyersburg	36 0	334	7	13	17	L	98	96	94	21	80	79	78
Greenville	35 5	1320	5	10	14	L	93	91	88	22	76	75	74
Jackson AP	35 4	413	8	14	17	L	97	95	94	21	80	79	78
Knoxville AP	35 5	980	9	13	17	L	95	92	90	21	77	76	75
Memphis AP	35 0	263	11	17	21	L	98	96	94	21	80	79	78
Murfreesboro	35 5	608	7	13	17	L	97	94	92	22	79	78	77
Nashville AP	36 1	577	6	12	16	L	97	95	92	21	79	78	77
Tullahoma	35 2	1075	7	13	17	L	96	94	92	22	79	78	77
TEXAS													
Ablene AP	32 3	1759	12	17	21	M	101	99	97	22	76	75	7

Climatic Conditions for United States and Canada (Continued)*-a

Col. 1 State and Station ^b	Col. 2 Latitude ^c	Col. 3 Elev. ^d ft	Winter				Summer								
			Median of Annual Ex- tremes	Col. 4		Col. 5 Coinci- dent Wind Ve- locity ^e	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range ^f	Col. 8 Design Wet-Bulb				
				99%	97½%		1%	2½%	5%		1%	2½%	5%		
				99%	97½%		1%	2½%	5%		1%	2½%	5%		
TEXAS (continued)															
Corpus Christi AP.....	27 5	43	28	32	36	M	95	93	91	19	81	80	80		
Corsicana.....	32 0	425	16	21	25	M	102	100	98	21	79	78	77		
Dallas AP.....	32 5	481	14	19	24	H	101	99	97	20	79	78	78		
Del Rio, Laughlin AFB.....	29 2	1072	24	28	31	M	101	99	98	24	79	77	76		
Denton.....	33 1	655	12	18	22	H	102	100	98	22	79	78	77		
Eagle Pass.....	28 5	743	23	27	31	L	106	104	102	24	80	79	78		
El Paso AP.....	31 5	3918	16	21	25	L	100	98	96	27	70	69	68		
Fort Worth AP.....	32 5	544r	14	20	24	H	102	100	98	22	79	78	77		
Galveston AP.....	29 2	5	28	32	36	M	91	89	88	10	82	81	81		
Greenville.....	33 0	575	13	19	24	H	101	99	97	21	79	78	78		
Harlingen.....	26 1	37	30	34	38	M	96	95	94	19	80	80	79		
Houston AP.....	29 4	50	23	28	32	M	96	94	92	18	80	80	79		
Houston CO.....	29 5	158r	24	29	33	M	96	94	92	18	80	80	79		
Huntsville.....	30 4	49r	22	27	31	M	99	97	96	20	80	79	78		
Killeen-Gray AFB.....	31 0	1021	17	22	26	M	100	99	97	22	78	77	76		
Lamesa.....	32 5	2965	7	14	18	M	100	98	96	26	74	73	72		
Laredo AFB.....	27 3	503	29	32	36	L	103	101	100	23	79	78	78		
Longview.....	32 2	345	16	21	25	M	100	98	96	20	81	80	79		
Lubbock AP.....	32 4	3243	4	11	15	M	99	97	94	26	73	72	71		
Lufkin AP.....	31 1	286	19	24	28	M	98	96	95	20	81	80	79		
McAllen.....	26 1	122	30	34	38	M	102	100	98	21	80	79	78		
Midland AP.....	32 0	2815r	13	19	23	M	100	98	96	26	74	73	72		
Mineral Wells AP.....	32 5	934	12	18	22	H	102	100	98	22	78	77	76		
Palestine CO.....	31 5	580	16	21	25	M	99	97	96	20	80	79	78		
Pampa.....	35 3	3230	0	7	11	M	100	98	95	26	73	72	71		
Pecos.....	31 2	2580	10	15	19	L	102	100	97	27	72	71	70		
Plainview.....	34 1	3400	3	10	14	M	100	98	95	26	73	72	71		
Port Arthur AP.....	30 0	16	25	29	33	M	94	92	91	19	81	80	80		
San Angelo, Goodfellow AFB.....	31 2	1878	15	20	25	M	101	99	97	24	76	75	74		
San Antonio AP.....	29 3	792	22	25	30	L	99	97	96	19	77	77	76		
Sherman-Perrin AFB.....	33 4	763	12	18	23	H	101	99	97	22	79	78	77		
Snyder.....	32 4	2325	9	15	19	M	102	100	97	26	75	74	73		
Temple.....	31 1	675	18	23	27	M	101	99	97	22	79	78	77		
Tyler AP.....	32 2	527	15	20	24	M	99	97	96	21	80	79	78		
Vernon.....	34 1	1225	7	14	18	H	103	101	99	24	77	76	75		
Victoria AP.....	28 5	104	24	28	32	M	98	96	95	18	80	79	79		
Waco AP.....	31 4	506	16	21	26	M	101	99	98	22	79	78	78		
Wichita Falls AP.....	34 0	994	9	15	19	H	103	100	98	24	77	76	75		
UTAH															
Cedar City AP.....	37 4	5613	-10	-1	6	VL	94	91	89	32	65	64	62		
Logan.....	41 4	4775	-7	3	7	VL	93	91	89	33	66	65	63		
Moab.....	38 5	3965	-2	12	16	VL	100	98	95	30	66	65	64		
Ogden CO.....	41 1	4400	-3	7	11	VL	94	92	89	33	66	65	64		
Price.....	39 4	5580	-7	3	7	VL	93	91	88	33	65	64	63		
Provo.....	40 1	4470	-6	2	6	L	96	93	91	32	67	66	65		
Richfield.....	38 5	5300	-10	-1	3	VL	94	92	89	34	66	65	64		
St. George CO.....	37 1	2899	-13	22	26	VL	104	102	99	33	71	70	69		
Salt Lake City AP.....	40 5	4220	-2	5	9	L	97	94	92	32	67	66	65		
Vernal AP.....	40 3	5280	-20	-6	-6	VL	90	88	84	32	64	63	62		
VERMONT															
Barre.....	44 1	1120	-23	-17	-13	L	86	84	81	23	73	72	70		
Burlington AP.....	44 3	331	-18	-12	-7	M	83	85	83	23	74	73	71		
Rutland.....	43 3	620	-18	-12	-8	L	87	85	82	23	74	73	71		
VIRGINIA															
Charlottesville.....	38 1	870	7	11	15	L	93	90	88	23	79	77	76		
Danville AP.....	36 3	590	9	13	17	L	95	92	90	21	78	77	76		
Fredericksburg.....	38 2	50	6	10	14	M	94	92	89	21	79	78	76		
Harrisonburg.....	38 3	1340	0	5	9	L	92	90	87	23	78	77	76		
Lynchburg AP.....	37 2	947	10	15	19	M	94	92	89	21	77	76	75		
Norfolk AP.....	36 5	26	18	20	23	M	94	91	89	18	79	78	78		
Petersburg.....	37 1	194	10	15	18	L	96	94	91	20	80	79	78		

Climatic Conditions for United States and Canada (Continued)*-a

Col. 1 State and Station ^b	Col. 2 Latitude ^c	Col. 3 Elev. ^d ft	Winter				Summer								
			Median of Annual Ex- tremes	Col. 4		Col. 5 Coinci- dent Wind Ve- locity ^e	Col. 6 Design Dry-Bulb			Col. 7 Out- door Daily Range ^f	Col. 8 Design Wet-Bulb				
				99%	97½%		1%	2½%	5%		1%	2½%	5%		
				99%	97½%		1%	2½%	5%		1%	2½%	5%		
VIRGINIA (continued)															
Richmond AP.....	37 3	162	10	14	18	L	96	93	91	21	79	78	77		
Roanoke AP.....	37 2	1174r	9	15	18	L	92	91	89	23	76	75	74		
Staunton.....	38 2	1480	3	8	12	L	94	92	87	23	78	77	75		
Winchester.....	39 1	750	1	6	10	L	94	92	89	21	78	76	75		
WASHINGTON															
Aberdeen.....	47 0	12	19	24	27	M	83	80	77	16	62	61	60		
Bellingham AP.....	48 5	150	8	14	18	L	76	74	71	19	67	65	63		
Bremerton.....	47 3	162	17	24	29	L	85	81	77	20	68	66	65		
Ellensburg AP.....	47 0	1729	-5	2	6	VL	91	89	86	34	67	65	63		
Everett-Paine AFB.....	47 5	598	13	19	24	L	82	78	74	20	67	65	63		
Kennewick.....	46 0	392	4	11	15	VL	98	96	93	30	69	68	66		
Longview.....	46 1	12	14	20	24	L	88	86	83	30	68	66	65		
Moses Lake, Larson AFB.....	47 1	1183	-14	-7	-1	VL	96	93	90	32	65	64	63		
Olympia AP.....	47 0	190	15	21	25	L	85	83	80	32	67	65	63		
Port Angeles.....	48 1	99	20	26	29	M	75	73	70	18	60	58	57		
Seattle-Boeing Fld.....	47 3	14	17	23	27	L	82	80	77	24	67	65	64		
Seattle CO.....	47 4	14	22	28	32	L	81	79	76	19	67	65	64		
Seattle-Tacoma AP.....	47 3	386	14	20	24	L	85	81	77	22	66	64	63		
Spokane AP.....	47 4	2357	-5	-2	4	VL	93	90	87	28	66	64	63		
Tacoma-McChord AFB.....	47 1	350	14	20	24	L	85	81	78	22	68	66	64		
Walla Walla AP.....	46 1	1185	-5	12	16	VL	98	96	93	27	69	68	66		
Wenatchee.....	47 2	634	-2	5	9	VL	95	92	89	32	68	66	64		
Yakima AP.....	46 3	1061	-1	6	10	VL	94	92	89	36	69	67	65		
WEST VIRGINIA															
Beckley.....	37 5	2330	-4	0	6	L	91	88	86	22	74	73	72		
Bluefield AP.....	37 2	2850	1	6	10	L	88	86	83	22	74	73	72		
Charleston AP.....	38 2	939	1	9	14	L	92	90	88	20	76	75	74		
Clarksburg.....	39 2	977	-2	3	7	L	92	90	87	21	76	75	74		
Elkins AP.....	38 5	1970	-4	1	5	L	87	84	82	22	74	73	72		
Huntington CO.....	38 2	565r	4	10	14	L	95	93	91	22	77	76	75		
Marionburg AP.....	39 2	537	6	10	14	L	86	84	81	21	78	77	76		
Morgantown AP.....	39 4	1245	-2	2	7	L	90	88	85	22	76	74	73		
Parkersburg CO.....	39 2	615r	2	8	12	L	93	91	88	21	77	76	75		
Wheeling.....	40 1	659	0	5	9	L	91	89	86	21	76	75	74		
WISCONSIN															
Appleton.....	44 2	742	-16	-10	-6	M	89	87	84	23	75	74	72		
Ashland.....	46 3	650	-27	-21	-17	L	85	83	80	23</					

Climatic Conditions for Other Foreign Countries

Col. 1 Country and Station	Col. 2 Latitude and Longitude	Col. 3 Elevation, Ft	Winter			Summer						
			Col. 4			Col. 5 Design Dry-Bulb			Col. 6 Out-door Daily Range F deg	Col. 7 Design Wet-Bulb		
			Mean of Annual Extremes	99%	97 1/2%	1%	2 1/2%	5%	1%	2 1/2%	5%	
ADEN	12 50N/45 02E	10	63	68	70	102	100	98	11	83	82	82
AFGHANISTAN												
Kabul	34 35N/69 12E	5955	2	6	9	98	96	93	32	66	65	64
ALGERIA												
Algiers	36 46N/3 03E	194	38	43	45	95	92	89	14	77	76	75
ARGENTINA												
Buenos Aires	34 35S/58 29W	89	27	32	34	91	89	86	22	77	76	75
Cordoba	31 22S/64 15W	1388	21	28	32	100	96	93	27	76	75	74
Treman	26 50S/65 10W	1401	24	32	36	102	99	96	23	76	75	74
AUSTRALIA												
Adelaide	34 56S/138 35E	140	36	38	40	98	94	91	25	72	70	68
Alice Springs	23 48S/133 53E	1795	28	34	37	104	102	100	27	75	74	72
Brisbane	27 28S/153 02E	137	39	44	47	91	88	86	18	77	76	75
Darwin	12 28S/130 51E	88	60	64	66	94	93	91	16	82	81	81
Melbourne	37 49S/144 58E	114	31	35	38	95	91	86	21	71	69	68
Perth	31 57S/115 51E	210	38	40	42	100	96	93	22	76	74	73
Sydney	33 52S/151 12E	138	38	40	42	89	84	80	13	74	73	72
AUSTRIA												
Vienna	48 15N/16 22E	644	2	6	11	88	86	83	16	71	69	67
AZORES												
Lajes (Terceira)	38 45N/27 05W	170	42	46	49	80	78	77	11	73	72	71
BAHAMAS												
Nassau	25 05N/77 21W	11	55	61	63	90	89	88	13	80	80	79
BELGIUM												
Brussels	50 48N/4 21E	328	13	15	19	83	79	77	19	70	68	67
BERMUDA												
Kindley AFB	33 22N/64 41W	129	47	53	55	87	86	85	12	79	78	78
BOLIVIA												
La Paz	16 30S/68 09W	12001	28	31	33	71	69	68	24	58	57	56
BRAZIL												
Belem	1 27S/48 29W	42	67	70	71	90	89	87	19	80	79	78
Belo Horizonte	19 56S/43 57W	3002	42	47	50	86	84	83	18	76	75	75
Brasilia	15 52S/47 55W	3442	46	49	51	89	88	86	17	76	75	75
Curitiba	25 25S/49 17W	3114	28	34	37	86	84	82	21	75	74	74
Fortaleza	3 46S/38 33W	89	66	69	70	91	90	89	17	79	78	78
Porto Alegre	30 02S/51 13W	33	32	37	40	95	92	89	20	76	76	75
Recife	8 04S/34 53W	97	67	69	70	88	87	86	10	78	77	77
Rio de Janeiro	22 55S/43 12W	201	56	58	60	94	92	90	11	80	79	78
Salvador	13 00S/38 30W	154	65	67	68	85	87	86	12	79	79	78
Sao Paulo	23 33S/46 38W	2608	36	42	46	86	84	82	18	75	74	74
BRITISH HONDURAS												
Belize	17 31N/88 11W	17	55	60	62	90	90	89	13	82	82	81
BULGARIA												
Sofia	42 42N/23 20E	1805	2	3	8	89	86	84	26	71	70	69
BURMA												
Mandalay	21 59N/96 06E	252	50	54	56	104	102	101	30	81	80	80
Rangoon	16 47N/96 09E	18	59	62	63	100	98	95	25	83	82	82
CAMBODIA												
Phnom Penh	11 33N/104 51E	36	62	66	68	98	96	94	19	83	82	82
CEYLON												
Colombo	6 54N/79 52E	24	65	69	70	90	89	88	15	81	80	80
CHILE												
Punta Arenas	53 10S/70 54W	26	22	25	27	68	66	64	14	56	55	54
Santiago	33 27S/70 42W	1706	27	32	35	90	89	88	32	71	70	69
Valparaiso	33 01S/71 35W	135	39	43	46	81	79	77	16	67	66	65
CHINA												
Chungking	29 33N/106 33E	755	34	37	39	99	97	95	18	81	80	79
Shanghai	31 12N/121 26E	23	16	23	26	94	92	90	16	81	81	80
COLOMBIA												
Baranquilla	10 59N/74 48W	44	66	70	72	95	94	93	17	83	82	82
Bogota	4 36N/74 05W	8406	42	45	46	72	70	69	19	60	59	58
Cali	3 25N/76 30W	3189	53	57	58	84	82	79	15	70	69	68
Medellin	6 13N/75 36W	4650	48	53	55	87	85	84	25	73	72	72
CONGO												
Brazzaville	4 15S/15 15E	1043	54	60	62	93	92	91	21	81	81	80
Kinasha (Leopoldville)	4 20S/15 13E	1066	54	60	62	92	91	90	19	81	80	80
Stanleyville	0 26N/15 14E	1370	65	67	68	92	91	90	19	81	80	80

Climatic Conditions for Other Foreign Countries (Continued)

Col. 1 Country and Station	Col. 2 Latitude and Longitude	Col. 3 Elevation, Ft	Winter			Summer													
			Col. 4			Col. 5 Design Dry-Bulb			Col. 6 Out-door Daily Range F deg	Col. 7 Design Wet-Bulb									
			Mean of Annual Extremes	99%	97 1/2%	1%	2 1/2%	5%	1%	2 1/2%	5%								
CUBA																			
Guantanamo Bay	19 54N/75 09W	21	60	64	66	94	93	92	16	82	81	80							
Havana	23 08N/82 21W	80	54	59	62	92	91	89	14	81	81	80							
CZECHOSLOVAKIA																			
Prague	50 05N/14 25E	662	3	4	9	88	85	83	16	66	65	64							
DENMARK																			
Copenhagen	55 41N/12 33E	43	11	16	19	79	76	74	17	68	66	64							
DOMINICAN REPUBLIC																			
Santo Domingo	18 29N/69 54W	57	61	63	65	92	90	88	16	81	80	80							
ECUADOR																			
Guayaquil	2 10S/79 53W	20	61	64	65	92	91	89	20	80	80	79							
Quito	0 13S/78 32W	9446	30	36	39	73	72	71	32	63	62	62							
EL SALVADOR																			
San Salvador	13 42N/89 13W	2238	51	54	56	98	96	95	32	77	76	75							
ETHIOPIA																			
Addis Ababa	9 02N/38 45E	7753	35	39	41	84	82	81	28	66	65	64							
Asmara	15 17N/38 55E	7028	36	40	42	83	81	80	27	65	64	63							
FINLAND																			
Helsinki	60 10N/24 57E	30	-11	-7	-1	77	74	72	14	66	65	63							
FRANCE																			
Lyon	45 42N/4 47E	938	-1	10	14	91	89	86	23	71	70	69							
Marseilles	43 18N/5 23E	246	23	25	28	90	87	84	22	72	71	69							
Nantes	47 15N/1 34W	121	17	22	26	86	83	80	21	70	69	67							
Nice	43 42N/7 16E	39	31	34	37	87	85	83	15	73	72	72							
Paris	48 49N/2 29E	164	16	22	25	89	86	83	21	70	68	67							
Strasbourg	48 35N/7 46E	465	9	11	16	86	83	80	20	70	69	67							
FRENCH GUIANA																			
Cayenne	4 56N/52 27W	20	69	71	72	92	91	90	17	83	83	82							
GERMANY																			
Berlin	52 27N/13 18E	187	6	7	12	84	81	78	19	68	67	66							
Hamburg	53 33N/9 58E	66	10	12	16	80	76	73	13	68	66	65							
Hannover	52 24N/9 40E	561	7	16	20	82	78	75	17	65	67	65							
Mannheim	49 34N/8 28E	359	2	8	11	87	85	82	18	71	69	68							
Munich	48 09N/11 34E	1729	-1	5	9	86	83	80	18	68	66	64							
GHANA																			
Accra	5 33N/0 12W	88	65	68	69	91	90	89	13	80	79	79							
GIBRALTAR																			
Gibraltar	36 09N/5 22W	11	38	42	45	92	89	86	14	76	75	74							
GREECE																			
Athens	37 58N/23 43E	351	29	33	36	96	93	91	18	72	71	71							
Thessaloniki	40 37N/22 57E	78	23	28	32	95	93	91	20	77	76	75							
GREENLAND																			
Narsarsuaq	61 11N/45 25W	85	-23	-12	-8	66	63	61	20	56	54	52							
GUATEMALA																			
Guatemala City	14 37N/90 31W	4855	45	48	51	83	82	81	24	69	68	67							
GUYANA																			
Georgetown	6 50N/58 12W	6	70	72	73	89	88	87	11	80	79	79							
HAITI																			
Port au Prince	18 33N/72 20W	121	63	65	67	97	95	93	20	82	81	80							
HONDURAS																			
Tegucigalpa	14 06N/87 13W	3094	44	47	50	89	87	85	28	73	72	71							
HONG KONG																			
Hong Kong	22 18N/114 10E	109	43	48	50	92	91	90	10	81	80	80							
HUNGARY																			

Climatic Conditions for Other Foreign Countries (Continued)

Col. 1 Country and Station	Col. 2 Latitude and Longitude	Col. 3 Elevation, Ft	Winter			Summer													
			Col. 4			Col. 5 Design Dry-Bulb			Col. 6 Out-door Daily Range F deg	Col. 7 Design Wet-Bulb									
			Mean of Annual Ex-tremes	99%	97½%	1%	2½%	5%		1%	2½%	5%							
IRAN																			
Abadan	30 21N/48 16E	7	32	39	41	116	113	110	32	82	81	81							
Meshed	36 17N/59 36E	3104	3	10	14	99	96	93	29	68	67	66							
Tehran	35 41N/51 25E	4002	15	20	24	102	100	98	27	75	74	73							
IRAQ																			
Baghdad	33 20N/44 24E	111	27	32	35	113	111	108	34	73	72	72							
Mosul	36 19N/43 09E	730	23	29	32	114	112	110	40	73	72	72							
IRELAND																			
Dublin	53 22N/6 21W	155	19	24	27	74	72	70	16	65	64	62							
Shannon	52 41N/8 55W	8	19	25	28	76	73	71	14	65	64	63							
ISRAEL																			
Jerusalem	31 47N/35 13E	2485	31	36	38	95	94	92	24	70	69	69							
Tel Aviv	32 06N/34 47E	36	33	39	41	96	93	91	16	74	73	72							
ITALY																			
Milan	45 27N/09 17E	341	12	18	22	89	87	84	20	76	75	74							
Naples	40 53N/14 18E	220	28	34	36	91	88	86	19	74	73	72							
Rome	41 48N/12 36E	377	25	30	33	94	92	89	24	74	73	72							
IVORY COAST																			
Abidjan	5 19N/4 01W	65	64	67	69	91	90	88	15	83	82	81							
JAPAN																			
Fukuoka	33 35N/130 27E	22	26	29	31	92	90	89	20	82	80	79							
Sapporo	43 04N/141 21E	56	7	1	5	86	83	80	20	76	74	72							
Tokyo	35 41N/139 46E	19	21	26	28	91	89	87	14	81	80	79							
JORDAN																			
Amman	31 57N/35 57E	2548	29	33	36	97	94	92	25	70	69	68							
KENYA																			
Nairobi	1 16S/36 48E	5971	45	48	50	81	80	78	24	66	65	65							
KOREA																			
Pyongyang	39 02N/125 41E	186	-10	-2	3	89	87	85	21	77	76	76							
Seoul	37 34N/126 58E	285	-1	7	9	91	89	87	16	81	79	78							
LEBANON																			
Beirut	33 54N/35 28E	111	40	42	45	93	91	90	15	78	77	76							
LIBERIA																			
Monrovia	6 18N/10 48W	75	64	68	69	90	89	88	19	82	82	81							
LIBYA																			
Bengasi	32 06N/20 04E	82	41	46	48	97	94	91	13	77	76	75							
MADAGASCAR																			
Tananarive	18 55S/47 33E	4531	39	43	46	86	84	83	23	73	72	71							
MALAYSIA																			
Kuala Lumpur	3 07N/101 42E	127	67	70	71	94	93	92	20	82	82	81							
Penang	5 25N/100 19E	17	69	72	73	93	93	92	18	82	81	80							
Singapore	1 18N/103 50E	33	69	71	72	92	91	90	14	82	81	80							
MARTINIQUE																			
Fort de France	14 37N/61 05W	13	62	64	66	90	89	88	14	81	81	80							
MEXICO																			
Guadalajara	20 41N/103 20W	5105	35	39	42	93	91	89	29	68	67	66							
Mérida	20 58N/89 38W	72	56	59	61	97	95	94	21	80	79	77							
Mexico City	19 24N/99 12W	7575	33	37	39	83	81	79	25	61	60	59							
Monterrey	25 40N/100 18W	1732	31	38	41	98	95	93	20	79	78	77							
Vera Cruz	19 12N/96 08W	184	55	60	62	91	89	88	12	83	83	82							
MOROCCO																			
Casablanca	33 35N/7 39W	164	36	40	42	94	90	86	50	73	72	70							
NEPAL																			
Katmandu	27 42N/85 12E	4388	30	33	35	89	87	86	25	78	77	76							
NETHERLANDS																			
Amsterdam	52 23N/4 55E	5	17	20	23	79	76	73	10	65	64	63							
NEW GUINEA																			
Manokwari	0 52S/134 05E	62	70	71	72	89	88	87	12	82	81	81							
Point Moresby	9 29S/147 09E	126	62	67	69	92	91	90	14	80	80	79							
NEW ZEALAND																			
Auckland	36 51S/174 46E	140	37	40	42	78	77	76	14	67	66	65							
Christ Church	43 32S/172 37E	32	25	28	31	82	79	76	17	68	67	66							
Wellington	41 17S/174 46E	394	32	35	37	76	74	72	14	66	65	64							
NICARAGUA																			
Managua	12 10N/86 15W	135	62	65	67	94	93	92	21	81	80	79							
NIGERIA																			
Lagos	6 27N/3 24E	10	67	70	71	92	91	90	12	82	82	81							
NORWAY																			
Bergen	60 24N/5 19E	141	14	17	20	75	74	73	21	67	66	65							
Oslo	59 56N/10 44E	308	-2	0	4	79	77	74	17	67	66	64							

Climatic Conditions for Other Foreign Countries (Continued)

Col. 1 Country and Station	Col. 2 Latitude and Longitude	Col. 3 Elevation, Ft	Winter			Summer													
			Col. 4			Col. 5 Design Dry-Bulb			Col. 6 Out-door Daily Range F deg	Col. 7 Design Wet-Bulb									
			Mean of Annual Ex-tremes	99%	97½%	1%	2½%	5%		1%	2½%	5%							
PAKISTAN																			
Chittagong	22 21N/91 50E	87	48	52	54	93	91	89	20	82	81	81							
Karachi	24 48N/66 59E	13	45	49	51	100	98	95	14	82	82	81							
Lahore	31 35N/74 20E	702	32	35	37	109	107	105	27	83	82	81							
Peshwar	34 01N/71 35E	1164	31	35	37	109	106	103	29	81	80	79							
PANAMA AND CANAL ZONE																			
Panama City	8 58N/79 33W	21	69	72	73	93	92	91	18	81	81	80							
PARAGUAY																			
Asunción	25 17S/57 30W	456	35	43	46	100	98	96	24	81	81	80							
PERU																			
Lima	12 05S/77 03W	394	51	53	55	86	85	84	17	76	75	74							
PHILIPPINES																			
Manila	14 35N/120 59E	47	69	73	74	94	92	91	20	82	81	81							
POLAND																			
Kraków	50 04N/19 57E	723	-2	2	6	84	81	78	19	68	67	66							
Warsaw	52 13N/21 02E	394	-3	3	8	84	81	78	19	71	70	68							
PORTUGAL																			
Lisbon	38 43N/9 08W	313	32	37	39	89	86	83	16	69	68	67							
PUERTO RICO																			
San Juan	18 29N/66 07W	82	65	67	68	89	88	87	11	81	80	79							
RUMANIA																			
Bucharest	44 25N/26 06E	269	-2	3	8	93	91	89	26	72	71	70							
SAUDI ARABIA																			
Dhahran	26 17N/50 09E	80	39	45	48	111	110	108	32	86	85	84							
Jedda	21 28N/39 10E	20	52	57	60	106	103	100	22	85	84	83							
Riyadh	24 39N/46 42E	1938	29	37	40	110	108	106	32	78	77	76							
SENEGAL																			
Dakar	14 42N/17 29W	131	58	61	62	95	93	91	13	81	80	80							
SOMALIA																			

Table C2

Earth Temperature Tables

for

Underground Heat Distribution System Design

The following Tables TG-1 through TG-11 were developed by applying monthly average temperatures prepared by the U. S. Weather Bureau for many localities in the United States to a technique described in the 1965 ASHRAE technical paper entitled "Earth Temperature and Thermal Diffusivity at Selected Stations in the United States" by T. Kusuda and P. R. Achenbach. These temperature data are, however, for the undisturbed earth. The earth temperature immediately under the building may be estimated by taking an arithmetic average of the building temperature and the design earth temperature found in the appropriate table. For example, the floor on grade in the Washington, D. C. area may be treated as a slab of 12" thickness with ground temperature of 70.5 °F if the room temperature is 75 °F and the summer design TG is determined from the data of Upper Marlboro, Maryland for ALPHA = 0.025 which is 66 °F.

Table TG-1

DRY SOIL

		T6				
		AVERAGE EARTH TEMPERATURE IN DEG. F.				
		THERMAL DIFFUSIVITY IN FT**2/HR				ALPHA = .010
STATION	STATES	WINTER	SPRING	SUMMER	FALL	YEAR
AUDURN,	ALABAMA	60.	61.	71.	70.	65.
DECATUR,	ALABAMA	52.	54.	65.	65.	59.
PALMER AACS,	ALASKA	31.	31.	42.	41.	36.
TEMPE,	ARIZONA	62.	64.	73.	74.	68.
TUCSON,	ARIZONA	68.	69.	77.	79.	73.
BRAWLEY,	CALIFORNIA	70.	73.	83.	84.	77.
DAVIS,	CALIFORNIA	61.	61.	72.	72.	67.
FT. COLLINS,	COLO.	44.	45.	58.	56.	51.
STORRS,	CONN.	46.	45.	58.	58.	52.
GAINESVILLE,	FLA.	65.	70.	77.	77.	73.
ATHENS,	GEORGIA	59.	61.	72.	72.	66.
MOSCOW,	IDAHO	43.	42.	52.	52.	47.
LEMONT,	ILLINOIS	46.	45.	59.	59.	52.
URBANA,	ILLINOIS	46.	47.	61.	60.	53.
WEST LAFAYETTE,	IND	47.	47.	62.	61.	54.
AMES,	IOWA	44.	45.	62.	60.	52.
BURLINGTON,	IOWA	47.	49.	66.	65.	56.
CASTANA,	IOWA	42.	42.	61.	59.	51.
COUNCIL BLUFFS,	IOWA	47.	47.	62.	62.	55.
SARATOGA,	IOWA	41.	40.	59.	57.	49.
SPENCER,	IOWA	42.	42.	58.	57.	50.
GARDEN CITY,	KANSAS	48.	51.	66.	66.	58.
MANHATTAN,	KANSAS	48.	50.	64.	64.	56.
MOUND VALLEY,	KANSAS	52.	54.	68.	68.	60.
LEXINGTON,	KENTUCKY	51.	52.	65.	64.	58.
UPPER MARLBORO,	MD.	48.	49.	63.	63.	56.
EAST LANSING,	MICH.	45.	43.	57.	57.	50.
FAIRMONT,	MINNESOTA	42.	43.	58.	57.	50.
FARIBAULT,	MINNESOTA	40.	40.	55.	53.	47.
ST. PAUL,	MINNESOTA	42.	40.	57.	56.	49.
WASECA,	MINNESOTA	41.	46.	59.	54.	50.
STATE UNIV.,	MISS.	60.	62.	73.	73.	67.
FAUCETT,	MISSOURI	47.	47.	61.	61.	54.
KANSAS CITY,	MO.	48.	49.	62.	61.	55.
SIKESTON,	MISSOURI	52.	54.	67.	67.	60.
SPICKARD,	MISSOURI	50.	49.	60.	62.	55.
BOZEMAN,	MONTANA	39.	37.	50.	48.	43.
HUNTLEY,	MONTANA	44.	44.	58.	57.	50.
LINCOLN,	NEBRASKA	45.	45.	60.	60.	53.
NE. BRUNSWICK,	N.J.	48.	48.	60.	60.	54.
ITHACA,	NEW YORK	44.	43.	54.	54.	49.
COLUMBUS,	OHIO	47.	47.	59.	60.	53.
COSHOCTON,	OHIO	46.	46.	58.	58.	52.
WOOSTER,	OHIO	46.	46.	58.	58.	52.
BARNSDALL,	OKLAHOMA	56.	57.	69.	69.	63.
LAKE HEFNER,	OKLA.	56.	57.	70.	71.	64.
PANMUSKA,	OKLAHOMA	54.	55.	68.	68.	61.
OTTAWA,	ONTARIO	42.	39.	54.	52.	47.
CORVALLIS,	OREGON	50.	51.	61.	60.	56.
HOOD RIVER,	OREGON	46.	48.	57.	57.	52.

AVERAGE EARTH TEMPERATURE IN DEG. F, TG

THERMAL DIFFUSIVITY IN FT**2/HR

ALPHA= .010

STATION	STATES	WINTER	SPRING	SUMMER	FALL	YEAR
MEUFORD, OREGON		51.	52.	61.	61.	56.
PENDLETON, OREGON		46.	49.	61.	60.	54.
STATE COLLEGE, PA.		46.	45.	59.	58.	52.
KINGSTON, R. I.		45.	43.	55.	56.	50.
CALHOUN, S. CAROLINA		56.	58.	70.	69.	63.
MADISON, S. DAKOTA		40.	40.	54.	54.	47.
JACKSON, TENNESSEE		53.	55.	66.	64.	59.
TEMPLE, TEXAS		64.	65.	77.	77.	71.
SALT LAKE CITY, UTAH		44.	45.	56.	55.	50.
BURLINGTON, VERMONT		42.	40.	54.	53.	48.
PULLMAN, WASHINGTON		43.	46.	55.	52.	50.
SEATTLE, WASHINGTON		48.	50.	56.	56.	53.
AFTON, WYOMING		43.	43.	53.	53.	48.

Table TG-2
AVERAGE SOIL

AVERAGE EARTH TEMPERATURE IN DEG. F, TG

THERMAL DIFFUSIVITY IN FT**2/HR

ALPHA= .025

STATION	STATES	WINTER	SPRING	SUMMER	FALL	YEAR
AUBURN, ALABAMA		57.	61.	74.	70.	65.
DECATUR, ALABAMA		49.	53.	69.	66.	59.
PALMER, ALASKA		29.	30.	45.	41.	36.
TEMPE, ARIZONA		58.	63.	77.	74.	68.
TUCSON, ARIZONA		65.	69.	80.	80.	73.
BRAWLEY, CALIFORNIA		66.	73.	87.	85.	77.
DAVIS, CALIFORNIA		57.	60.	76.	73.	67.
FT. COLLINS, COLO.		40.	44.	62.	57.	51.
STORRS, CONN.		43.	44.	62.	59.	52.
GAINESVILLE, FLA.		61.	71.	79.	78.	73.
ATHENS, GEORGIA		55.	60.	75.	73.	66.
MOSCOW, IDAHO		40.	42.	55.	53.	47.
LEMONT, ILLINOIS		42.	44.	64.	60.	52.
URBANA, ILLINOIS		42.	47.	65.	61.	53.
WEST LAFAYETTE, IND		43.	47.	66.	62.	54.
AMES, IOWA		39.	44.	67.	61.	52.
BURLINGTON, IOWA		42.	48.	71.	66.	56.
CASTANA, IOWA		36.	41.	66.	61.	51.
COUNCIL BLUFFS, IOWA		42.	47.	67.	63.	55.
SARATOGA, IOWA		37.	39.	64.	58.	49.
SPENCER, IOWA		37.	41.	62.	58.	50.
GARDEN CITY, KANSAS		42.	51.	71.	67.	58.
MANHATTAN, KANSAS		44.	49.	68.	65.	56.
MOUND VALLEY, KANSAS		47.	54.	72.	69.	60.
LEXINGTON, KENTUCKY		47.	51.	69.	65.	58.
UPPER MARLBORO, MD.		44.	49.	66.	64.	56.
EAST LANSING, MICH.		41.	41.	61.	58.	50.
FAIRMONT, MINNESOTA		38.	43.	63.	57.	50.
FARIBAULT, MINNESOTA		36.	38.	59.	54.	47.
ST. PAUL, MINNESOTA		38.	38.	62.	57.	49.
WASECA, MINNESOTA		36.	47.	64.	54.	50.
STATE UNIV., MISS.		56.	62.	76.	74.	67.
FAUCETT, MISSOURI		43.	45.	65.	61.	54.
KANSAS CITY, MO.		44.	48.	65.	62.	55.
SIKESTON, MISSOURI		48.	54.	72.	68.	60.
SPICKARD, MISSOURI		47.	48.	63.	64.	55.
BOZEMAN, MONTANA		36.	36.	53.	49.	43.
HUNTLEY, MONTANA		40.	43.	63.	57.	50.
LINCOLN, NEBRASKA		40.	44.	65.	61.	53.
NEW BRUNSWICK, N.J.		44.	47.	63.	61.	54.
ITHACA, NEW YORK		41.	41.	58.	54.	49.
COLUMBUS, OHIO		43.	46.	63.	61.	53.
CUSHOCTON, OHIO		42.	45.	61.	59.	52.
WOOSTER, OHIO		42.	45.	62.	59.	52.
BARNSDALL, OKLAHOMA		53.	56.	73.	70.	63.
LAKE HEPNER, OKLA.		52.	56.	74.	72.	64.
PAWBUKA, OKLAHOMA		50.	54.	72.	68.	61.
OTTAWA, ONTARIO		39.	37.	58.	52.	47.
CORVALLIS, OREGON		47.	50.	64.	60.	55.
HOOD RIVER, OREGON		43.	48.	59.	57.	52.

AVERAGE EARTH TEMPERATURE IN DEG. F: TG

THERMAL DIFFUSIVITY IN FT**2/HR

ALPHA= .025

STATION STATES	WINTER	SPRING	SUMMER	FALL	YEAR
MEUFORD, OREGON	48.	52.	64.	61.	56.
PENNYLETON, OREGON	41.	49.	65.	61.	54.
STATE COLLEGE, PA.	42.	44.	63.	59.	52.
KINGSTON, R. I.	41.	41.	58.	57.	50.
CALHOUN, S. CAROLINA	52.	57.	73.	70.	63.
MADISON, S. DAKOTA	36.	38.	59.	55.	47.
JACKSON, TENNESSEE	50.	55.	69.	64.	59.
TEMPLE, TEXAS	61.	65.	81.	77.	71.
SALT LAKE CITY, UTAH	40.	45.	60.	56.	50.
BURLINGTON, VERMONT	39.	38.	59.	54.	48.
PULLMAN, WASHINGTON	40.	45.	58.	52.	50.
SEATTLE, WASHINGTON	46.	50.	59.	56.	53.
AFTON, WYOMING	41.	42.	56.	53.	48.

Table TG-3

WET SOIL

AVERAGE EARTH TEMPERATURE IN DEG. F: TG

THERMAL DIFFUSIVITY IN FT**2/HR

ALPHA= .050

STATION STATES	WINTER	SPRING	SUMMER	FALL	YEAR
AUBURN, ALABAMA	54.	61.	76.	70.	65.
DECATUR, ALABAMA	46.	53.	71.	65.	59.
PALMER, ALASKA	27.	30.	48.	41.	36.
TEMPE, ARIZONA	56.	64.	79.	74.	68.
TUCSON, ARIZONA	62.	69.	82.	81.	73.
BRAWLEY, CALIFORNIA	63.	73.	90.	84.	77.
DAVIS, CALIFORNIA	55.	60.	78.	73.	67.
FT. COLLINS, COLO.	37.	45.	65.	56.	51.
STORRS, CONN.	40.	44.	65.	59.	52.
GAINESVILLE, FLA.	58.	72.	81.	79.	73.
ATHENS, GEORGIA	52.	61.	78.	73.	66.
MOSCOW, IDAHO	38.	42.	57.	53.	47.
LEMONT, ILLINOIS	39.	44.	67.	60.	52.
URBANA, ILLINOIS	39.	47.	68.	60.	53.
WEST LAFAYETTE, IND	40.	47.	69.	62.	54.
AMES, IOWA	35.	44.	70.	61.	52.
BURLINGTON, IOWA	38.	48.	74.	66.	56.
CASTANA, IOWA	32.	42.	70.	61.	51.
COUNCIL BLUFFS, IOWA	39.	47.	70.	63.	55.
SARATOGA, IOWA	33.	39.	68.	58.	49.
SPEINGER, IOWA	33.	42.	66.	58.	50.
GARDEN CITY, KANSAS	38.	52.	74.	67.	58.
MANHATTAN, KANSAS	40.	49.	72.	65.	56.
MOUND VALLEY, KANSAS	44.	55.	75.	69.	60.
LEXINGTON, KENTUCKY	44.	51.	72.	65.	58.
UPPER MERIDORE, MD.	41.	49.	69.	64.	56.
EAST LANSING, MICH.	39.	41.	64.	57.	50.
FAIRMONT, MINNESOTA	35.	43.	67.	57.	50.
FARIBAULT, MINNESOTA	34.	38.	62.	54.	47.
ST. PAUL, MINNESOTA	35.	38.	65.	57.	49.
WASECA, MINNESOTA	31.	49.	67.	53.	50.
STATE UNIV., MISS.	53.	62.	78.	74.	67.
FAUCETT, MISSOURI	41.	45.	68.	61.	54.
KANSAS CITY, MO.	41.	48.	68.	61.	55.
SIKESTON, MISSOURI	45.	54.	75.	68.	60.
SPICKARD, MISSOURI	44.	48.	65.	64.	55.
BOZEMAN, MONTANA	34.	35.	57.	48.	43.
HUNTLEY, MONTANA	37.	43.	66.	57.	50.
LINCOLN, NEBRASKA	36.	44.	68.	62.	53.
NEW BRUNSWICK, N.J.	41.	47.	66.	62.	54.
ITHACA, NEW YORK	39.	41.	61.	54.	49.
COLUMBUS, OHIO	40.	47.	65.	61.	53.
CUSHOCTON, OHIO	40.	45.	64.	60.	52.
WOOSTER, OHIO	40.	45.	65.	59.	52.
BARNSDALL, OKLAHOMA	50.	56.	75.	70.	63.
LAKE HEFNER, OKLA.	49.	57.	77.	73.	64.
PAWBUKA, OKLAHOMA	48.	54.	75.	68.	61.
OTTAWA, ONTARIO	37.	37.	61.	51.	47.
CORVALLIS, OREGON	45.	51.	67.	60.	55.
HOOVER RIVER, OREGON	41.	49.	61.	57.	52.

AVERAGE EARTH TEMPERATURE IN DEG. F.		T _G				
THERMAL DIFFUSIVITY IN FT**2/HR		ALPHA = .050				
STATION	STATES	WINTER	SPRING	SUMMER	FALL	YEAR
MEDFORD	OREGON	46.	52.	66.	61.	56.
PENDLETON	OREGON	38.	50.	68.	60.	54.
STATE COLLEGE	PA.	40.	44.	66.	59.	52.
KINGSTON	R. I.	39.	41.	61.	57.	50.
CALHOUN	S. CAROLINA	49.	58.	76.	69.	63.
MADISON	S. DAKOTA	33.	38.	62.	55.	47.
JACKSON	TENNESSEE	48.	55.	72.	64.	59.
TEMPLE	TEXAS	58.	65.	84.	77.	71.
SALT LAKE CITY	UTAH	37.	45.	62.	55.	50.
BURLINGTON	VERMONT	37.	38.	62.	54.	48.
PULLMAN	WASHINGTON	37.	45.	60.	50.	50.
SEATTLE	WASHINGTON	44.	50.	60.	56.	53.
AFTON	WYOMING	39.	42.	59.	53.	48.

Table C3

Thermophysical Properties of Wall/Roof/Floor

Reprinted by permission from 1972 Handbook of Fundamentals (American Society of Heating, Refrigerating and Air-Conditioning Engineers, 345 East 47th Street, New York, New York), p. 431.

Thermal Properties and Code Numbers of Layers Used in Calculations of Coefficients for Wall and Roof Transfer Functions

Table C4

Description	Code Number	Thickness and Thermal Properties ^a				
		L	K	D	SH	R
Outside surface resistance	AO					0.333
1" stucco (asbestos cement or wood siding plaster, etc.)	A1	0.0833	0.4	116	0.20	
4" face brick (dense concrete)	A2	0.333	0.77	125	0.22	
Steel siding (aluminum or other light-weight cladding)	A3	0.005	26.0	490	0.10	
Outside surface resistance, 1/2" slag, membrane and 1" felt	A4	0.0417 0.0313	0.83 0.11	55 70	0.40 0.40	0.333
Outside surface resistance	A5					0.333
Finish	A6	0.0417	0.24	78	0.26	
Air space resistance	B1					0.91
1" insulation	B2	0.083	0.025	2.0	0.2	
2" insulation	B3	0.167	0.025	2.0	0.2	
3" insulation	B4	0.25	0.025	2.0	0.2	
1" insulation	B5	0.0833	0.025	5.7	0.2	
2" insulation	B6	0.167	0.025	5.7	0.2	
1" wood	B7	0.0833	0.07	37.0	0.6	
2.5" wood	B8	0.2083	0.07	37.0	0.6	
4" wood	B9	0.333	0.07	37.0	0.6	
2" wood	B10	0.167	0.07	37.0	0.6	
3" wood	B11	0.25	0.07	37.0	0.6	
3" insulation	B12	0.25	0.025	5.7	0.2	
4" clay tile	C1	0.333	0.33	70.0	0.2	
4" l.w. concrete block	C2	0.333	0.22	38.0	0.2	
4" h.w. concrete block	C3	0.333	0.47	61.0	0.2	
4" common brick	C4	0.333	0.42	120	0.2	
4" h.w. concrete	C5	0.333	1.0	140	0.2	
8" clay tile	C6	0.667	0.33	70	0.2	
8" l.w. concrete block	C7	0.667	0.22	38.0	0.2	
8" h.w. concrete block	C8	0.667	0.6	61.0	0.2	
8" common brick	C9	0.667	0.42	120	0.2	
8" h.w. concrete	C10	0.667	1.0	140	0.2	
12" h.w. concrete	C11	1.0	1.0	140	0.2	
2" h.w. concrete	C12	0.167	1.0	140	0.2	
6" h.w. concrete	C13	0.5	1.0	140	0.2	
4" l.w. concrete	C14	0.333	0.1	40	0.2	
6" l.w. concrete	C15	0.5	0.1	40	0.2	
8" l.w. concrete	C16	0.667	0.1	40	0.2	
Inside surface resistance	E0					0.685
1" plaster; 1/2" gypsum or other similar finishing layer	E1	0.0625	0.42	100	0.2	
1" slag or stone	E2	0.0417	0.83	55	0.40	
1" felt & membrane	E3	0.0313	0.11	70	0.40	
Ceiling air space	E4					1.0
Acoustic Tile	E5	0.0625	0.035	30	0.20	

^a Units: L = feet. K = Btu per (hr) (sq ft) (F deg). D = lb per cu ft. SH = Btu per (lb) (F deg). R = (hr) (sq ft) (F deg) per Btu.

Typical Watt/ft.² of floor area data

	<u>Lighting</u>	<u>Equipment</u>
Apartment	1.7	1.2
Office	5.0	1.0
Department Stores	4.0	0.0
School	5.0	0.0

Table C5
Shading Coefficients

Shading Coefficients for Single Glass and
Insulating Glass^a

A. Single Glass					
Type of Glass	Nominal Thickness ^b	Solar Trans. ^b	Shading Coefficient		
			$h_g = 4.0$	$h_g = 3.0$	
Regular Sheet Regular Plate/ Float	$\frac{1}{8}$, $\frac{1}{4}$	0.87	1.00	1.00	
	$\frac{3}{16}$	0.80	0.95	0.97	
	$\frac{1}{2}$	0.75	0.91	0.93	
	$\frac{3}{4}$	0.71	0.88	0.91	
Grey Sheet	$\frac{1}{8}$	0.59	0.78	0.80	
	$\frac{1}{4}$	0.74	0.90	0.92	
	$\frac{3}{16}$	0.45	0.66	0.70	
	$\frac{1}{2}$	0.71	0.88	0.90	
	$\frac{3}{4}$	0.67	0.86	0.88	
Heat-Absorbing Plate/Float ^d	$\frac{1}{8}$	0.52	0.72	0.75	
	$\frac{1}{4}$	0.47	0.70	0.74	
	$\frac{3}{16}$	0.33	0.56	0.61	
	$\frac{1}{2}$	0.24	0.50	0.57	

B. Insulating Glass ^a					
Type of Glass	Nominal Thickness ^c	Solar Trans. ^b		Shading Coefficient	
		Outer Pane	Inner Pane	$h_g = 4.0$	$h_g = 3.0$
Regular Sheet Out, Regular Sheet In	$\frac{1}{8}$, $\frac{1}{4}$	0.87	0.87	0.90	0.90
Regular Plate/Float Out, Regular Plate/Float In	$\frac{1}{4}$	0.80	0.80	0.83	0.83
Heat-Abs Plate/Float Out, Regular Plate/Float In	$\frac{1}{4}$	0.46	0.80	0.56	0.58

^a Refers to factory-fabricated units with $\frac{1}{4}$, $\frac{1}{2}$, or $\frac{3}{4}$ in. air space or to prime windows plus storm windows.
^b Refer to manufacturer's literature for values.
^c Thickness of each pane of glass, not thickness of assembled unit.
^d Refers to grey, bronze, and green tinted heat-absorbing plate/float glass.

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Shading Coefficients for Single Glass with Indoor Shading by Venetian Blinds and Roller Shades

Type of Glass	Nominal Thickness ^a	Solar Trans. ^b	Type of Shading				
			Venetian Blinds		Roller Shade		
			Medium	Light	Opaque		Translucent
					Dark	White	
Regular Sheet Regular Plate/Float Regular Pattern Heat-Absorbing Pattern Grey Sheet	$\frac{1}{8}$ to $\frac{1}{4}$ $\frac{1}{4}$ to $\frac{1}{2}$ $\frac{1}{4}$ to $\frac{3}{8}$ $\frac{1}{8}$ $\frac{1}{8}$, $\frac{3}{8}$	0.87-0.80 0.80-0.71 0.87-0.79 0.74, 0.71	0.64	0.55	0.59	0.25	0.39
Heat-Absorbing Plate/Float ^d Heat-Absorbing Pattern Grey Sheet	$\frac{1}{8}$, $\frac{1}{4}$ $\frac{1}{8}$, $\frac{1}{4}$ $\frac{1}{8}$, $\frac{3}{8}$	0.46 — 0.59, 0.45	0.57	0.53	0.45	0.30	0.36
Heat-Absorbing Plate/Float or Pattern Heat-Absorbing Plate/Float ^d	— $\frac{1}{4}$	0.44-0.30 0.34	0.54	0.52	0.40	0.28	0.32
Heat-Absorbing Plate or Pattern	—	0.29-0.15 0.24	0.42	0.40	0.36	0.28	0.31
Reflective Coated Glass S.C. ^c = 0.30 0.40 0.50 0.60			0.25 0.33 0.42 0.50	0.23 0.29 0.38 0.44			

^a Refer to manufacturer's literature for values.
^b For vertical blinds with opaque white and beige louvers in the tightly closed position, SC is 0.25 and 0.29 when used with glass of 0.71 to 0.80 transmittance.
^c Shading Coefficient for glass with no shading device.
^d Refers to grey, bronze, and green tinted heat-absorbing plate/float glass.

Shading Coefficients for Insulating Glass^a with Indoor Shading by Venetian Blinds and Roller Shades

Type of Glass	Nominal Thickness, each light	Solar Trans. ^b		Type of Shading				
		Outer Pane	Inner Pane	Venetian Blinds ^c		Roller Shade		
				Medium	Light	Opaque		Translucent
		Dark	White					
Regular Sheet Out Regular Sheet In Regular Plate/Float Out Regular Plate/Float In	$\frac{1}{8}$, $\frac{1}{4}$ $\frac{1}{4}$	0.87 0.80	0.87 0.80	0.57	0.51	0.60	0.25	0.37
Heat-Absorbing Plate/Float ^d Out Regular Plate/Float In	$\frac{1}{4}$	0.46	0.80	0.39	0.36	0.40	0.22	0.30
Reflective Coated Glass SC ^c = 0.20 0.30 0.40				0.19 0.27 0.34	0.18 0.26 0.33			

^a Refers to factory-fabricated units with $\frac{1}{8}$, $\frac{1}{4}$, or $\frac{1}{2}$ in. air space, or to prime windows plus storm windows.
^b Refer to manufacturer's literature for exact values.
^c For vertical blinds with opaque white or beige louvers, tightly closed, SC is approximately the same as for opaque white roller shades.
^d Refers to bronze or green tinted heat-absorbing plate/float glass.
^e Shading Coefficient for glass with no shading device.

Properties of Representative Indoor Shading

Indoor Shade	Solar Properties (Normal Incidence)		
	Trans.	Reflect.	Absorp.
Venetian Blinds ^a (Ratio of slat width to slat spacing 1.2, slat angle 45 deg)			
Light Colored Slat	0.05	0.55	0.40
Medium Colored Slat	0.05	0.35	0.60
Vertical Blinds			
White Louvers	0.00	0.77	0.23
Roller Shades			
Light Shades (Translucent)	0.25	0.60	0.15
White Shade (Opaque)	0.00	0.80	0.20
Dark Colored Shade (Opaque)	0.00	0.12	0.88

^a The values shown in this table and preceding tables are based on horizontal Venetian blinds. However, tests show these values may be used for vertical blinds with good accuracy.

Shading Coefficients for Double Glazing with Between-Glass Shading

Type of Glass	Nominal Thickness, each pane	Solar Trans. ^a		Description of Air Space	Type of Shading		
		Outer Pane	Inner Pane		Venetian Blinds		Louvered Sun Screen
					Light	Medium	
Regular Sheet Out Regular Sheet In Regular Plate Out Regular Plate In	$\frac{1}{8}$, $\frac{1}{4}$ $\frac{1}{4}$	0.87 0.80	0.87 0.80	Shade in contact with glass or shade separated from glass by air space. Shade in contact with glass-voids filled with plastic.	0.33 —	0.36 —	0.43 0.49
Heat-Abs. Plate/Float ^b Out Regular Plate In	$\frac{1}{4}$	0.46	0.80	Shade in contact with glass or shade separated from glass by air space. Shade in contact with glass-voids filled with plastic.	0.28 —	0.30 —	0.37 0.41

^a Refer to manufacturer's literature for exact values.
^b Refers to grey, bronze, and green tinted heat-absorbing plate/float glass.

Shading Coefficients for Single and Insulating Glass with Draperies

Glazing	Glass Trans.	Glass SC*	Shading Coefficient For Index Letters in Fig. 10**									
			A	B	C	D	E	F	G	H	I	J
Single Glass	0.80	0.95	0.35	0.40	0.45	0.50	0.55	0.60	0.65	0.70	0.75	0.80
in. Regular	0.71	0.85	0.35	0.39	0.43	0.48	0.52	0.56	0.61	0.66	0.70	0.74
in. Regular	0.46	0.67	0.33	0.36	0.38	0.41	0.44	0.46	0.49	0.52	0.54	0.57
in. Heat Abs.	0.34	0.57	0.32	0.34	0.36	0.38	0.41	0.43	0.45	0.47	0.49	0.51
in. Heat Abs.	0.24	0.50	0.30	0.32	0.33	0.34	0.36	0.38	0.39	0.40	0.42	0.43
Reflective Coated (See Manufacturers' literature for exact values)	—	0.60	0.35	0.36	0.38	0.41	0.43	0.46	0.49	0.51	0.54	0.57
	—	0.50	0.31	0.33	0.34	0.36	0.38	0.39	0.41	0.42	0.44	0.46
	—	0.40	0.26	0.27	0.28	0.29	0.30	0.32	0.33	0.34	0.35	0.36
	—	0.30	0.20	0.21	0.21	0.22	0.23	0.23	0.23	0.24	0.24	0.25
Insulating Glass (½ in. Air Space) Regular Out and Regular In	0.64	0.83	0.35	0.37	0.42	0.45	0.48	0.52	0.56	0.58	0.62	0.66
Heat Abs. Out and Regular In	0.37	0.56	0.32	0.33	0.35	0.37	0.39	0.41	0.43	0.45	0.47	0.49
Reflective Coated (see Manufacturers' literature for exact values)	—	0.40	0.28	0.28	0.29	0.31	0.32	0.34	0.36	0.37	0.37	0.38
	—	0.30	0.24	0.24	0.25	0.25	0.26	0.26	0.27	0.27	0.25	0.29
	—	0.20	0.15	0.15	0.16	0.16	0.17	0.17	0.18	0.18	0.19	0.19

* For glass alone, with no drapery.
** Shading coefficient values for the SC lines in Fig. 10 for representative glazings. Substitute for the SC index letters in Fig. 10 the values on the line of the glazing selected.

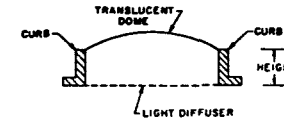


Fig. Terminology for Domed Skylights

Table 23 Shading Coefficients for Domed Skylights

Dome	Light Diffuser (Translucent)	Curb (See Fig. 1)		Shading Coefficient	U-Value
		Height, in.	Width to Height Ratio		
Clear τ=0.86	yes τ=0.58	0	∞	0.61	0.46
		9	5	0.56	0.43
		18	2.5	0.50	0.40
Clear τ=0.86	None	0	∞	0.99	0.80
		9	5	0.88	0.75
		18	2.5	0.80	0.70
Translucent τ=0.52	None	0	∞	0.57	0.80
		9	5	0.51	0.75
		18	2.5	0.46	0.70
Translucent τ=0.27	None	0	∞	0.34	0.80
		9	5	0.30	0.75
		18	2.5	0.28	0.70

Shading Coefficients for Louvered Sun Screens

Profile Angle, deg	Group 1		Group 2	
	Transmittance	SC	Transmittance	SC
10	0.23	0.35	0.25	0.33
20	0.06	0.17	0.14	0.23
30	0.04	0.15	0.12	0.21
40 and above	0.04	0.15	0.11	0.20

Profile Angle, deg	Group 3		Group 4	
	Transmittance	SC	Transmittance	SC
10	0.40	0.51	0.48	0.59
20	0.32	0.42	0.39	0.50
30	0.21	0.31	0.28	0.38
40 and above	0.07	0.18	0.20	0.30

Group 1. Black, width over spacing ratio 1.15/1, 23 louvers per inch.
Group 2. Light color, high reflectance, otherwise same as Group 1.
Group 3. Black or dark color, w/s ratio 0.85/1, 17 louvers per inch.
Group 4. Light color or unpainted aluminum, high reflectance, otherwise same as Group 3.
U-value = 0.85 Btu/(sq ft)(F deg) for all groups when used with single glazing.

Shading Coefficients for Hollow Glass Block Wall Panels*

Type of Glass Block ^b	Description of Glass Block	Shading Coefficient ^c	
		Panels ^d in the Sun	Panels ^e in the Shade (N, NW, W, SW)
Type I	Glass Colorless or Aqua Smooth Face A, D: Smooth B, C: Smooth or wide ribs, or flutes horizontal or vertical, or shallow configuration. E: None	0.65	0.40
Type IA	Same as Type I except A: Ceramic Enamel on exterior face.	0.27	0.20
Type II	Same as Type I except E: Glass fiber screen.	0.44	0.34
Type III	Glass Colorless or Aqua A, D: Narrow vertical ribs or flutes. B, C: Horizontal light-diffusing prisms, or horizontal light-directing prisms. E: Glass fiber screen.	0.33	0.27
Type IIIA	Same as Type III except E: Glass fiber screen with green ceramic spray coating, or glass fiber screen and gray glass, or glass fiber screen with light-selecting prisms.	0.25	0.18

* For glass block used in horizontal skylights see Tables 28 and 29, Chapter 26 of the 1963 ASHRAE GUIDE AND DATA BOOK.
^b All values are for 7½ × 7½ × 3½ in. block, set in light-colored mortar. For 11½ × 11½ × 3½ in. block increase coefficients by 15 percent, and for 5½ × 5½ × 3½ in. blocks reduce coefficients by 15 percent.
^c Shading coefficients are to be applied to Heat Gain Factors for one hour earlier than the time for which the load calculation is made to allow for heat storage in the panel.
^d Shading coefficients are for peak load condition, but provide a close approximation for other conditions. For more precise values for other conditions, see Reference 20.
^e For NE, E, and SE panels in the shade add 50 percent to the values listed for panels in the shade.

Table C6

Absorptivity of Materials to Solar Radiation

BUILDING MATERIALS, SOLAR ABSORPTIVITY	
Material	Solar Absorptivity
BRICKS	
Clay, cream, glazed	0.36
Clay, Felton, dark portion	0.63
Clay, Felton, light portion	0.40
Lime clay, French	0.46
Gault, cream	0.36
Light buff	0.516
Light buff but darker than above	0.60
Mottled purple	0.77
Red	0.699
Red, common and tiles	0.68
Red, darker, glazed	0.766
Red, wire-cut	0.52
Stafford blue	0.89
Stock, light fawn	0.57
White glazed	0.26
White glazed (2 specimens)	0.25-0.27
TILES	
Clay, purple (dark)	0.82
Clay, dark purple, machine-made	0.81
Red	0.67
Red, hand-made	0.60
Red, light, Dutch	0.43
Red, light, machine-made	0.66
Red, light, machine-made	0.62
Concrete, uncolored	0.65
Concrete, black	0.91
Concrete, dark	0.91
Concrete, brown	0.85
Concrete, brown, very rough	0.88

(Continued on next page)

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BUILDING MATERIALS, SOLAR ABSORPTIVITY (Continued)

Material	Solar Absorptivity
ASPHALT	
New, 3 specimens	0.91
New, 3 specimens	0.91
New, another specimen	0.93
Pavement	0.852
Pavement, free from dust	0.928
Pavement, weathered, 3 specimens	0.82
	0.83
	0.89
ROOFING	
Bituminous felt, aluminized	0.40
Bituminous felt	0.88
Bituminous felt	0.89
Bitumin-covered, brown	0.87
Sheet, green	0.86
Sheet, black matte surface	0.97
Sheet, black matte surface	0.97
ASBESTOS CEMENT	
Aged	0.75
Aged 6 months	0.61
Aged 12 months	0.71
Aged 6 years, very dirty	0.83
Red	0.69
Red	0.74
Washed with soap and water	0.40
White	0.61
White (2 samples)	0.49-0.42
LIMESTONE	
Anston	0.60
Bath	0.53
Clipsham	0.46
Indiana	0.571
Ketton	0.42
Portland	0.36
Steetley	0.33
SAND-LIME	
Light-red	0.55
Red	0.68
White, fine sand	0.41
White, coarse sand	0.50
MARBLE	
White	0.44
Ground, unpolished	0.465
Cleavage	0.592
GRANITE	
Reddish	0.55
FELDSPAR	
K ₂ O Al ₂ O ₃ 6SiO ₂	0.606
MORTAR SCREENED	
	0.73

(Continued on next page)

BUILDING MATERIALS, SOLAR ABSORPTIVITY (Continued)

Material	Solar Absorptivity
SANDSTONE	
Grey, Bristol pennant	0.76
Polmaise, light fawn	0.54
Stancliffe, light grey	0.62
Woolton, red	0.73
WHITEWASH	
On galvanized iron	0.22
On galvanized iron	0.22
On galvanized iron	0.26
On galvanized iron, a very thick layer	0.20
SLATE	
Blue grey	0.87
Blue, grey	0.85
Clay, dark	0.933
Greensish, grey, rough	0.88
Grey, dark	0.90
Grey, dark, fairly rough	0.90
Grey, dark, fairly rough	0.90
Grey, dark, smooth	0.89
Purple	0.86
Silver-grey, Norwegian	0.79

BUILDING MATERIALS, SOLAR ABSORPTIVITY

Building Materials	Solar Absorptivity
Thickly tinned surface	0.05
Wood, smoothly planed	0.78
Basalt	0.72
Red sandstone	0.60
Marble (white)	0.58
Granite	0.45
Dolomite lime	0.41
Clay shale	0.69
Paris plaster	0.78
White plastered wall	0.92
Gravel	0.29
Sand	0.76
Glass	0.93
Sawdust	0.75
Clay	0.39
Red brick wall	0.93

CLOTH, SOLAR REFLECTIVITY

Material	Solar Reflectivity
QM1, cotton sheeting bleached, 4 oz per yd	0.62-0.66
QM2, cotton sateen prepared for dyeing, 9 oz per yd	0.68-0.72
QM4, cotton sateen undyed, 9 oz per yd	0.69-0.72
QM6, cotton sateen, medium gray, 9 oz per yd	0.53
QM7, cotton sateen dark gray, 9 oz per yd	0.24
50 percent wool, 50 percent cotton knit, undyed, 10.5 oz per yd	0.62
Cotton knit, undyed, 3 oz per yd	0.60

10. Appendix D

Fortran Listing of NBSLD

Although the attached Fortran listing of the NBSLD routine basically embodies the algorithms of Appendix A, some of the subroutines are considerably simplified if compared with the exact adaptation of Appendix A.

It is cautioned also that the Fortran used herein is the UNIVAC version of Fortran V, which is somewhat different from the ANSI standard Fortran.

PARACHUTE CLOTH, SOLAR ABSORPTIVITY, REFLECTIVITY, AND TRANSMISSIVITY

Material	Absorptivity	Reflectivity	Transmissivity
Dacron, 100 lb	0.05	0.35	0.60
Dacron, 300 lb	0.11	0.54	0.35
Dacron, 600 lb	0.12	0.61	0.27
Dacron, 800 lb	0.19	0.62	0.19
Nylon rip-stop (orange) 1.1 oz per sq yd, MIL-C-7020B Type I	0.13	0.23	0.64
Nylon rip-stop 1.1 oz per sq yd (white) MIL-C-7020	0.08	0.27	0.65
Nylon rip-stop 1.6 oz per sq yd (white) MIL-C-7020B Type III	0.06	0.22	0.72
Nylon cloth 2.25 oz per sq yd, MIL-C-7350B Type I	0.05	0.36	0.59
Nylon cloth 4.30 oz per sq yd, MIL-C-8021 Type I	0.08	0.44	0.48
Nylon cloth 7.0 oz per sq yd, MIL-C-8021 Type II	0.13	0.46	0.41
Nylon cloth 14.0 oz per sq yd, MIL-C-8021 Type III	0.11	0.62	0.27

```

1 @FCR.IS A1
2 C NBSLD IS A RESEARCH PROGRAM THAT IS CONTINUALLY CHANGED AND UPDATED.
3 C .....NBSLD.....
4 C
5 C
6 C NBSLD IS A RESEARCH PROGRAM OF NBS FOR THE PURPOSE OF
7 C STUDYING HEATING AND COOLING LOAD AND ROOM TEMPERATURE
8 C OF BUILDING UNDER ACTUAL WEATHER CONDITION
9 C A(I) AREA OF SURFACE I, FT2
10 C ABSP(I) SOLAR HEAT ABSORPTION COEFFICIENT FOR SURFACE I.
11 C THIS DATA REQUIRED FOR OPAQUE SURFACES ONLY.
12 C AENDW AREA OF THE ATTIC END WALL, FT2
13 C AG GROUND HEAT TRANSFER AREA, FT2 (MAY=0.)
14 C AIRCHG NO. OF ATTIC AIR CHANGES PER HR, DAYTIME
15 C AIRNT ATTIC NIGHT TIME AIR CHANGE MULTIPLIER
16 C ARCHGS NO. OF AIR CHANGES PER HR IN SUMMER
17 C ARCHGW NO. OF AIR CHANGES PER HR IN WINTER
18 C ATCACG NO. OF ATTIC AIR CHANGES PER HR (DAY OF NIGHT)
19 C AVEHTG AVERAGE HOURLY HEAT GAIN ENTIRE BUILDING, BTU/HR
20 C AZW(I) WALL AZIMUTH ANGLE FOR SURFACE I, DEGREES
21 C SOUTH = 0.
22 C WEST = 90.
23 C NORTH = 180.
24 C EAST = -90.
25 C BLDMAX BUILDING MAXIMUM SENSIBLE HEAT GAIN, BTU/HR
26 C CFML SUMMER INFILTRATION RATE, FT3/MIN.
27 C CFMV VENTILATION RATE, FT3/MIN.
28 C CFMWT WINTER INFILTRATION, FT3/MIN.
29 C CLDAY DAILY TCTAL ENERGY CONSUMPTION FOR A GRUOP
30 C (NCRM OF THEM) OF ROOMS OF THE SAME CONFIG-
31 C URATION, BTU
32 C CLDSUM RUNNING TOTAL ENERGY CONSUMPTION FOR COOLING OF
33 C ALL THE ROOMS IN A BUILDING OVER A SET TIME
34 C PERIOD, BTU
35 C CN CLEARNESS NUMBER
36 C CR(L) RESPONSE FACTOR COMMON RATIO FOR CONSTRUCTION L
37 C DAY DAY OF YEAR
38 C DAYSKP NO. OF DAYS TO BE SKIPPED FROM THE WEATHER TAPE
39 C (FROM ITS LAST STARTING POSITION)
40 C DB(J) OUTDOOR DRYBULB TEMPERATURE AT HOUR J, F
41 C DBA DAILY AVERAGE OUTSIDE DRYBULB TEMPERATURE, F
42 C DBIN DESIGN INDOOR DRYBULB TEMPERATURE, F
43 C DBM DRYBULB MEAN, F
44 C DBMAX DESIGN OUTDOOR MAXIMUM DRYBULB TEMPERATURE, F
45 C DBMWT DESIGN WINTER OUTDOOR DRYBULB TEMPERATURE, F
46 C DBNBS(J) FRACTION OF RANGE TO USE FOR DESIGN PROFILE
47 C AT HOUR J
48 C DP OUTDOOR DEW POINT, F
49 C DPID INDOOR DEW POINT, F
50 C DPIN DESIGN INDOOR DEW POINT, F
51 C DR(L) RESPONSE FACTOR COMMON RATIO FOR CONSTRUCTION L
52 C (SAME AS CR(L))
53 C DST DAYLIGHT SAVING TIME INDICATOR
54 C ELAPS DAYS ELAPSED SINCE JANUARY 1
55 C G(IV,VI) RADIATION CONFIGURATION FACTORS FOR
56 C RADIATION FROM SURFACE VI TO SURFACE IV
57 C H(I) EXTERIOR SURFACE HEAT TRANSFER COEFFICIENT
58 C FOR SURFACE I, BTU/HR,FT2,F

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59 C HI(I) INTERIOR SURFACE CONVECTION HEAT TRANSFER
60 C COEFFICIENT, BTU/HR,FT2.F
61 C HIND INDOOR ENTHALPY FOR DESIGN CONDITIONS, BTU/LB
62 C HLDAY DAILY TOTAL ENERGY CONSUMPTION FOR HEATING FOR A
63 C GROUP (NORM OF THEM) OF ROOMS OF THE SAME
64 C CONFIGURATION, BTU
65 C HLDSUM RUNNING TOTAL ENERGY CONSUMPTION FOR HEATING OF
66 C ALL THE ROOMS IN A BUILDING, BTU
67 C HOUT OUTDOOR ENTHALPY FOR DESIGN CONDITIONS, BTU/LB
68 C HR INNER SURFACE RADIATIVE HEAT TRANSFER COEFFICIENT
69 C (=4.*(535.**3)*SIGMA)
70 C HT CEILING HEIGHT, FT
71 C IHT(I) HEAT TRANSFER INDEX
72 C =-1 FOR GLASS SURFACE
73 C = 0 OPAQUE
74 C = 1 OTHERWISE
75 C IMAX HOUR OF DAY FOR MAXIMUM COOLING LOAD
76 C INCLUD =0 INCLUDE ROOM IN SUMMARY
77 C =1 OTHERWISE
78 C IRF(I) RESPONSE FACTOR INDEX FOR SURFACE I
79 C IROT DEGREES OF ROTATION
80 C ISKIP = 1 SKIP RESPONSE FACTOR CALCULATION
81 C AND BUILDING DATA INPUT
82 C = 0 OTHERWISE
83 C ITK AND ITHST INDICES FOR ROOM TEMPERATURE COMPUTATION
84 C ITK=0, ITHST=1 ROOM TEMP PRESCRIBED, EITHER CONSTANT
85 C OR WITH NIGHT TIME SET-BACK
86 C ITK=1, ITHST=0 ROOM TEMP NOT BEING CONTROLLED. NO A/C.
87 C ITK=1, ITHST=1 ROOM TEMP FLOAT WITHIN PRESCRIBED UPPER
88 C AND LOWER LIMITS. NO A/C WHEN WITHIN
89 C THE LIMITS.
90 C ITK=0, ITHST=0 EQUIPMENT CAPACITY PRESCRIBED. ROOM TEMP
91 C FLOAT WITHIN PRESCRIBED UPPER AND LOWER
92 C LIMITS, AND WHEN EQUIPMENT CAPACITY IS
93 C EXCEEDED.
94 C ITYPE(I) TYPE OF SURFACE I
95 C = 1 ROOF
96 C = 2 EXPOSED WALL
97 C = 3 WINDOW
98 C = 4 DOOR
99 C = 5 GROUND HEAT TRANSFER SURFACE
100 C = 6 INTERNAL MASS, FURNISHINGS, PARTY WALLS,
101 C PARTITION WALLS, AND FLOOR/CEILINGS
102 C = 7 OPEN PASSAGES
103 C = 8 EXPOSED FLOOR (EXPOSED UNDERSIDE)
104 C LAT LATITUDE, DEGREES
105 C LONG LONGITUDE, DEGREES
106 C LPYR LEAP YEAR INDICATOR
107 C MONTH MONTH OF YEAR
108 C MR(L) NUMBER OF RESPONSE FACTOR TERMS GENERA-
109 C TED BY RESPTK FOR CONSTRUCTION L
110 C SAME AS NR(L)
111 C NAMEBD NAME OF ROOM
112 C NE NUMBER OF SURFACES IN EAST WALL
113 C NEXP TOTAL NUMBER OF SURFACES IN ROOM
114 C = 2+NS+NW+NN+NE
115 C NMAX HR OF THE DAY WHEN QLMAX OCCURS
116 C NN NUMBER OF SURFACES IN NORTH WALL

117 C NOFLR NUMBER OF FLOORS
118 C NORM NO. OF ROOMS HAVING THE SAME DATA
119 C NR(L) NUMBER OF RESPONSE FACTOR TERMS CALCU-
120 C LATED BY RESPTK FOR CONSTRUCTION L
121 C NS NUMBER OF SURFACES IN SOUTH WALL
122 C NW NUMBER OF SURFACES IN WEST WALL
123 C PB BAROMETRIC PRESSURE
124 C = 29.921 INCHES OF MERCURY
125 C PI = 3.1415...
126 C PV VAPOR PRESSURE, INCHES OF MERCURY
127 C QCU MAXIMUM NUMBER OF OCCUPANTS
128 C QDES(I) HEAT GAIN OF SURFACE I AT HOUR IMAX, BTU/HR
129 C QDESIN(I,J) HEAT GAIN OF SURFACE I AT HOUR J,
130 C BTU/HR
131 C QEOPD EQUIPMENT MAXIMUM HEAT LOAD, BTU/HR
132 C QEOPX MAXIMUM EQUIPMENT LOAD, WATTS/FT2
133 C QEOP(J) EQUIPMENT LOAD AT HOUR J, BTU/HR
134 C QEQUX(J) EQUIPMENT USE SCHEDULE
135 C QGLAS(I,J) HEAT GAIN OF GLASS FOR I AT HOUR J,
136 C BTU/HR
137 C QGX(I) HEAT TRANSMISSION OF GLASS FOR SURFACE
138 C I AT HOUR IMAX, BTU/HR
139 C QI(I) INSIDE SURFACE HEAT FLUX OF SURFACE I,
140 C BTU/HR,FT2
141 C QISAVE(J,I) INSIDE SURFACE HEAT FLUX OF SURFACE
142 C I AT HOUR J, BTU/HR,FT2
143 C QLDS SUM OF LATENT AND SENSIBLE LOAD AT HOUR
144 C J, BTU/HR
145 C QLITE(J) LIGHT LOAD AT HOUR J, BTU/HR
146 C QLITO MAXIMUM LIGHT LOAD, BTU/HR
147 C QLITX(J) LIGHT USE SCHEDULE
148 C QLITY MAXIMUM LIGHTING LOAD, WATTS/FT2
149 C QLMAX ABSOLUTE VALUE OF THE MAX COOLING (OR
150 C HEATING) LOAD OF THE DAY, BTU/HR
151 C QLL(J) LATENT HEAT LOAD AT HOUR J, BTU/HR
152 C QLS(J) SENSIBLE HEAT LOAD AT HOUR J, BTU/HR
153 C QO(I) OUTSIDE SURFACE HEAT FLUX OF SURFACE I,
154 C BTU/HR, FT2
155 C QOCPS(J) OCCUPANT LOAD AT HOUR J, BTU/HR
156 C QOCUP(J) OCCUPANT SCHEDULE
157 C QPEOPL(J) PEOPLE LATENT LOAD AT HOUR J, BTU/HR
158 C QPLX MAX OCCUPANT LATENT LOAD, BTU/HR,PERSON
159 C QPSX MAX OCCUPANT SENSIBLE LOAD,BTU/HR,PERSON
160 C QSAVE(M,J) HEAT GAINS AND LOADS AT HOUR J, BTU/HR
161 C M = 1 TIME, HR
162 C M = 2 SENSIBLE HEAT GAIN, BTU/HR
163 C M = 3 LATENT HEAT GAIN, BTU/HR
164 C M = 4 SENSIBLE LOAD, BTU/HR
165 C M = 5 TOTAL LOAD, BTU/HR
166 C QSKY(I,J) HEAT RADIATED TO SKY BY SURFACE I T
167 C HOUR J, BTU/HR, FT2
168 C QSUMT SUM OF TOTAL HEAT GAINS FOR 24 HOURS,
169 C BTU/HR
170 C QSUN(I,J) INCIDENT SOLAR RADIATION FOR SURFACE I
171 C AT HOUR J, BTU/HR,FT2
172 C QTL(J) LATENT HEAT GAIN FROM INFILTRATION AT
173 C HOUR J, BTU/HR
174 C QWINT HEAT LOSS IN WINTER, BTU/HR

175	C	RANGE	DAILY RANGE OF OUTDOOR DRYBULB, F	233	C	TV	TEMPERATURE OF VENTILATING AIR, F
176	C	RHIN	DESIGN INDOOR RELATIVE HUMIDITY	234	C	TVN	TIME ZONE NUMBER
177	C	RHOUT	DESIGN OUTDOOR RELATIVE HUMIDITY	235	C	U(I)	OVERALL HEAT TRANSFER COEFFICIENT FOR SURFACE I
178	C	RCOMNO	RCCM NUMBER	236	C		
179	C	S	INFORMATION ARRAY REQUIRED BY SUBROUTINE SUN AND GLASS	237	C	UCELNG	OVERALL HEAT TRANSFER COEFFICIENT OF THE CEILING BETWEEN THE ATTIC AIR AND THE ROOM AIR BELOW
180	C			238	C		
181	C	SHADE(I)	SHADING COEFFICIENT FOR SURFACE I	239	C		
182	C	SIGMA	= 0.1714E-8	240	C	UENDW	OVERALL HEAT TRANSFER COEFFICIENT OF THE ATTIC ENDWALL
183	C	SITELD(J)	OVERALL COOLING LOAD AT HOUR J, BTU/HR	241	C		
184	C	SITEQL(J)	OVERALL LATENT HEAT GAIN AT HOUR J, BTU/HR	242	C	UG	GROUND HEAT TRANSFER COEFFICIENT
185	C			243	C	UGLAS	WINTER GLASS HEAT TRANSFER COEFFICIENT
186	C	SITEQS(J)	OVERALL SENSIBLE HEAT GAIN AT HOUR J, BTU/HR	244	C	UT(I)	U VALUE WITHOUT SURFACE RESISTANCES
187	C			245	C	VIN	INDOOR AIR SPECIFIC VOLUME, FT3/LB
188	C	SITETH(J)	OVERALL TOTAL HEAT GAIN AT HOUR J, BTU/HR	246	C	VOUT	OUTDOOR AIR SPECIFIC VOLUME, FT3/LB
189	C			247	C	VT(L)	SAME AS UT(I)
190	C	SITMAX	OVERALL MAXIMUM HEAT GAIN, BTU/HR	248	C	WA	OUTDOOR AIR HUMIDITY RATIO, LB OF H2O VAPOR PER LB OF DRY AIR (= WOUT)
191	C	SOTHTX	OVERALL HEAT GAIN AT HOUR IMAX, BTU/HR	249	C		
192	C	SOLLDD	TOTAL COOLING LOAD, BTU/HR	250	C	WAZ(I)	WALL AZIMUTH ANGLE MEASURED CLOCKWISE FROM SOUTH, DEGREES
193	C	SQWINT	OVERALL TOTAL HEAT LOSS, BTU/HR	251	C		
194	C	TA	ROOM AIR TEMPERATURE, F	252	C	WBID	DESIGN INDOOR WETBULB TEMPERATURE, F
195	C	TASAVE(J)	RCCM AIR TEMPERATURE AT HOUR J, F	253	C	WJMAX	DESIGN OUTDOOR WETBULB TEMPERATURE, F
196	C	TCLLD	DAILY RUNNING TOTAL ENERGY CONSUMPTION FOR COOLING FOR A GROUP (NORM OF THEM) OF ROOMS HAVING THE SAME CONFIGURATION, BTU	254	C	WBSAVE(J)	INDOOR WETBULB TEMPERATURE AT HOUR J, F
197	C			255	C	WID	DESIGN INDOOR HUMIDITY RATIO, LB OF H2O VAPOR/LB OF DRY AIR
198	C			256	C		
199	C			257	C	WIN	INDOOR HUMIDITY RATIO, LB H2O/LB DRY AIR
200	C	TG	DESIGN SUMMER GROUND TEMPERATURE, F	258	C	WOUT	DESIGN OUTDOOR HUMIDITY RATIO, LB H2O VAPOR/LB DRY AIR
201	C	TGW	DESIGN WINTER GROUND TEMPERATURE, F	259	C		
202	C	THTLDD	DAILY RUNNING TOTAL ENERGY CONSUMPTION FOR HEATING FOR A GROUP (NORM OF THEM) OF ROOMS HAVING THE SAME CONFIGURATION, BTU	260	C	WROT	DEGREES OF ROTATION FOR ROOM
203	C			261	C	WT	WALL TILT ANGLE (= 90. DEGREES WHEN VERTICAL WALL)
204	C			262	C		
205	C			263	C	WV	VENTILATION AIR HUMIDITY RATIO, LB H2O VAPOR/LB DRY AIR
206	C	TI(J)	INSIDE SURFACE TEMPERATURE RELATIVE TO THE REFERENCE TEMPERATURE AT HOUR J	264	C		
207	C			265	C	X(L,N)	RESPONSE FACTORS FOR CONSTRUCTION L
208	C	TIF(J)	INSIDE SURFACE TEMPERATURE AT HOUR J, F	266	C	XX(N,L)	TRANSPOSE OF ARRAY X
209	C	TIFSAV(J,I)	INSIDE SURFACE TEMPERATURE OF SURFACE I AT HOUR J, F	267	C	Y(L,N)	RESPONSE FACTORS FOR CONSTRUCTION L
210	C			268	C	YY(N,L)	TRANSPOSE OF ARRAY Y
211	C	TIM	INDOOR DESIGN MEAN (REFERENCE TEMPERATURE), F	269	C	Z(L,N)	RESPONSE FACTORS FOR CONSTRUCTION L
212	C			270	C	ZBLDG	INPUT ARRAY FOR BUILDING AND EXTERNAL DATA
213	C	TID	INDOOR DESIGN TEMPERATURE, F	271	C	ZROOM	INPUT ARRAY FOR ROOM DATA
214	C	TIS(I,J)	INSIDE SURFACE TEMPERATURE RELATIVE TO THE REFERENCE TEMPERATURE OF SURFACE I AT HOUR J, F	272	C	ZZ(N,L)	TRANSPOSE OF ARRAY Z
215	C			273	C		
216	C			274	C		
217	C	TIX(J)	INDOOR DESIGN DRYBULB TEMPERATURE AT HOUR J, F	275	C		
218	C			276	C		
219	C	TNEW(I)	UPDATED OUTSIDE SURFACE TEMPERATURE OF SURFACE I AT EVERY TIME INCREMENT, F	277	C		
220	C			278	C		
221	C	TNSAV(J,I)	UPDATED OUTSIDE SURFACE TEMPERATURE OF SURFACE I AT HOUR J, F	279	C		
222	C			280	C		
223	C	TOS(I,J)	OUTSIDE SURFACE TEMPERATURE RELATIVE TO REFERENCE TEMPERATURE OF SURFACE I AT HOUR J, F	281	C		
224	C			282	C		
225	C			283	C		
226	C	TOTHTX	TOTAL COOLING LOAD FOR A ROOM, BTU/HR	284	C		
227	C	TOY(J)	ARRAY USED FOR TEMPORARY STORAGE OF VALUES WHILE ADVANCING TEMPERATURE AS REQUIRED BY RESPONSE FACTOR METHOD	285	C		
228	C			286	C		
229	C	TSAVE	MAXIMUM TOTAL COOLING LOAD, BTU/HR	287	C		
230	C			288	C		
231	C	TSIGHT	TOTAL OVERALL HEAT GAIN FOR 24 HOURS, BTU/HR	289	C		
232	C			290	C		

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291     DIMENSION DBPF(24) / .87,.92,.96,.99,
292     2 1.00,.98,.93,.84,.71,.56,.39,.23,.11,.03,
293     3 0.0,.03,.10,.21,.34,.47,.58,.68,.76,.82/
294     COMMON /SHDW/ SHAW(30,15)
295     DIMENSION SHDX(20),SHDF(30,24),AIRLK(24),QSOL(24)
296     DIMENSION V(15),PLAT(24),AIRLAT(24),RALD(24),BASEL(24)
297     INTEGER DSTX,DSTY,RUNID,RUNTYP,ASHRAE
298     REAL LAT, LONG, NOFLR
299     INTEGER YEAR, TAPE2
300     LOGICAL LL1, LL2, LL3
301     COMMON /SOL/ LAT, LONG, TZN, WAZ, WT, CN, DSX, LPYR, S(35)
302     COMMON NSKP
303     PI=3.1415927
304     WRITE (6,1790)
305     WRITE (6,1800)
306     WRITE (6,1810)
307     WRITE (6,1820)
308     WRITE (6,1270)
309     READ (5,1460) RUNID,RUNTYP,ASHRAE,IDETAL,METHOD
310     READ (5,1640) NAMEBD
311     WRITE (6,1630) NAMEBD
312 C     READ 24 HOUR PROFILES FOR LIGHTING,EQUIPMENT AND OCCUPANCY
313     J3=3
314     DO 10 J=1,J3
315     IF (J.EQ.1) WRITE (6,1280)
316     IF (J.EQ.2) WRITE (6,1290)
317     IF (J.EQ.3) WRITE (6,1300)
318     READ (5,1460) (QLITX(I,J),I=1,24)
319     IF (J.EQ.1) WRITE (6,1310)
320     IF (J.EQ.2) WRITE (6,1320)
321     IF (J.EQ.3) WRITE (6,1330)
322     READ (5,1460) (QEQUX(I,J),I=1,24)
323     IF (J.EQ.1) WRITE (6,1340)
324     IF (J.EQ.2) WRITE (6,1350)
325     IF (J.EQ.3) WRITE (6,1360)
326     READ (5,1460) (QOCUP(I,J),I=1,24)
327 10    CONTINUE
328     IF (RUNTYP.GT.2) RUNTYP=2
329     WRITE (6,1370)
330     READ (5,1460) RMDBS
331     WRITE (6,1380)
332     READ (5,1460) RMDBW
333     WRITE (6,1390)
334     READ (5,1460) RMDBWD,RMDBSD,RHW,RHS
335     SIGMA=0.1714E-8
336     HR=4.*0.9*SIGMA*(530.**3)
337     WRITE (6,1400)
338     READ (5,1460) NDAY,NSKIP,TAPE2
339     WRITE (6,1410)
340     READ (5,1460) ZBLDG
341     CLDSUM=0.
342     HLDSUM=0.
343     DO 1260 IJKL,MN=1,10
344     WRITE (6,1420)
345     READ (5,1630) NAMERM
346 C     IF TAPE2 IS NOT BLANK B TAPE SHOULD BE ASSIGNED
347     IF (NAMERM(1).EQ.4H ) STOP
348     WRITE (6,1430)

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349     READ (5,1460) IROT,ISKIP,INCLUD
350 C     IF ISKIP .NE. 0, RESPONSE FACTOR CALCULATION IS SKIPPED
351 C     SC NO WALL DATA IS NEEDED
352     IF (ISKIP.NE.0) GO TO 50
353     DO 20 I=1,10
354     DO 20 J=1,100
355     X(I,J)=0.
356     Y(I,J)=0.
357 20    Z(I,J)=0.
358     IF (RUNID.EQ.1) GO TO 30
359     READ (8) X,Y,Z,MR,DR,VT
360     GO TO 40
361 C     THIS RESPONSE FACTOR ROUTINE REQUIRES MANY CONSTRUCTION DATA
362 C     PLEASE REFER TO THE INPUT INSTRUCTIONS
363 30    CONTINUE
364     CALL RESFX (X,Y,Z,XX,YY,ZZ,MR,DR,VT,10)
365     WRITE (8) X,Y,Z,MR,DR,VT
366     END FILE 8
367 40    PB=29.921
368 50    IF (IROT.NE.0) GO TO 60
369     WROT=0.
370     WRITE (6,1440)
371     READ (5,1460) ZROOM
372     WRITE (6,1450)
373     READ (5,1460) IW,IL,ISTART,ILEAVE
374     READ (5,1460) TUL,TLL,QCMAX,QHMAX,DBVMAX,DBVMIN
375     READ (5,1460) ITHST,ITK
376     CALL ROOMX (NEXP,NS,NW,NN,NE,HT)
377     ROOMNC=ZROOM(1)
378     MONTH=ZBLDG(1)
379     AG=A(NEXP)
380     NOFLR=1
381     QCU=ZROOM(4)
382     LAT=ZBLDG(12)
383     LONG=ZBLDG(11)
384     TZN=ZBLDG(13)
385     DAYSKP=NSKIP
386     QLITY=ZROOM(2)
387     QEGPX=ZROOM(3)
388     CFMV=ZROOM(8)
389     WEMAX=ZBLDG(6)
390     FLCG=ZROOM(5)
391     TGS=ZBLDG(8)
392     TGW=ZBLDG(9)
393     LDAY=ZBLDG(2)
394     YEAR=2000
395     DBWT=ZBLDG(7)
396     DBMAX=ZBLDG(4)
397     RHOW=ZBLDG(15)
398     ZLF=ZBLDG(14)
399     DBIN=RMDBW(12)
400     RHIN=RHW
401     IF (RUNTYP.EQ.2) CALL PSY1 (DBMAX,WBMAX,PB,DPMAX,PV,WA,HA,VA,RHO)
402     UG=ZBLDG(10)
403     TV=ZROOM(7)
404     FRAS=ZROOM(6)
405     ZNORM=ZROOM(12)
406     ARCHGW=ZROOM(10)

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407 CFMWT=AG*HT*ARCHGW/60.
408 ARCHGM=ZROOM(11)
409 CFMIN=AG*HT*ARCHGM/60.
410 CONST=ARCHGW/0.695
411 C THESE AIR CHANGE VALUES ARE FOR THE ATTIC VENTILATION
412 C ROOM AIR CHANGE VALUES WILL BE DETERMINED AS A FUNCTION OF
413 C WIND SPEED AND TEMPERATURE DIFFERENTIAL
414 CONTINUE
415 60 WRITE (6,1510)
416 IF (IDETAL.EQ.0) GO TO 70
417 WRITE (6,1550)
418 WRITE (6,1560) RCDMND,HT,AG,NOFLR,QCU,ZROOM(9),ZROOM(10)
419 WRITE (6,1740)
420 70 CONTINUE
421 S(1)=LAT
422 S(2)=LONG
423 S(3)=TZN
424 IF (IDETAL.EQ.0) GO TO 80
425 WRITE (6,1570)
426 WRITE (6,1560) LAT, LONG, TZN, ZNORM
427 WRITE (6,1740)
428 WRITE (6,1580)
429 RHIN=RHS
430 WRITE (6,1560) QLITY,QEQPX,CFMV,DBIN,TGW,TV,RHIN
431 WRITE (6,1740)
432 WRITE (6,1600) NEXP,ITK,ITHST
433 80 CONTINUE
434 WRITE (6,1510)
435 IF (IROT.NE.0) GO TO 90
436 WRITE (6,1470)
437 READ (5,1460) UENDW,UCELNG,AENDW,ATCMT,AIRCHG,AIRNT
438 WRITE (6,1480)
439 READ (5,1460) IEXTSD,IEXMS,IEXME,NTVNT,NVENT
440 CFMNT=NTVNT*AG*HT/60.
441 90 CONTINUE
442 IF (IDETAL.EQ.0) GO TO 100
443 WRITE (6,1740)
444 WRITE (6,1590)
445 WRITE (6,1560) UENDW,UCELNG,AENDW,ATCMT
446 100 CONTINUE
447 IF (IROT.NE.0) GO TO 210
448 SUM=0.
449 DO 200 I=1,NEXP
450 K=IRF(I)
451 IF (Y(K,1).GT.1.) IRF(I)=10
452 NR(I)=NR(K)
453 IF (IRF(I).EQ.10) NR(I)=1
454 UT(I)=VT(K)
455 CR(I)=DR(K)
456 IF (NR(I).EQ.0) NR(I)=1
457 IF (NR(I).GT.48) NR(I)=48
458 IF (ITYPE(I).EQ.3) ABSP(I)=0.
459 IF (ITYPE(I).EQ.5) ABSP(I)=0.
460 IF (ITYPE(I).GE.6) ABSP(I)=0.
461 IHT(I)=1
462 IF (ITYPE(I).EQ.3) IHT(I)=-1
463 HI(I)=6.0
464 HI(I)=1.46-HR

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465 IF (ITYPE(I).GE.5) H(I)=0.
466 IF (ITYPE(I).EQ.1) H(I)=1.630-HR
467 IF (ITYPE(I).EQ.5) H(I)=1.080-HR
468 IF (ITYPE(I).EQ.7) U(I)=500.
469 IF (ITYPE(I).EQ.8) H(I)=1.46
470 IF (IRF(I).NE.10) U(I)=0.
471 IF (U(I)) 110,110,120
472 110 RU=1./UT(I)+1./HI(I)+HR
473 IF (ITYPE(I).LT.5.OR.ITYPE(I).EQ.8) RU=RU+1./H(I)
474 U(I)=1./RU
475 120 CONTINUE
476 IF (X(K,2)) 170,130,170
477 130 IF (H(I)) 140,150,140
478 140 R=1./U(I)-1./H(I)
479 GO TO 160
480 150 R=1./U(I)
481 160 UT(I)=1./(R-1./HI(I)+HR)
482 IF (UT(I).LE.0.) UT(I)=28.0
483 IF (ITYPE(I).EQ.7) UT(I)=500.
484 170 CONTINUE
485 IF (UCELNG) 190,190,180
486 180 IF (ITYPE(I).NE.1) GO TO 190
487 RTA=1./UCELNG-1./HI(I)+HR
488 UT(I)=1./RTA
489 190 CONTINUE
490 UW(I)=U(I)
491 IF (ITYPE(I).GT.4) GO TO 200
492 SUM=SUM+A(I)*U(I)
493 200 CONTINUE
494 IF (ZLF.EG.0.) ZLF=1.
495 ZK=SUM/ZLF
496 FC=1.-0.02*ZK
497 210 IF (IROT.EQ.0) GO TO 240
498 WROT=IROT
499 DO 220 I=1,NEXP
500 AZW(I)=AZW(I)+WROT
501 IF (AZW(I).LT.-180.) AZW(I)=AZW(I)+360.
502 220 IF (AZW(I).GT.180.) AZW(I)=AZW(I)-360.
503 DO 230 I=1,NEXP
504 DO 230 J=1,NEXP
505 230 G(I,J)=G(I,J)/HR
506 240 CONTINUE
507 IF (IDETAL.EQ.0) GO TO 280
508 WRITE (6,1680)
509 DO 250 I=1,NEXP
510 WRITE (6,1650) I,ITYPE(I),IHT(I),IRF(I),ABSP(I),U(I),H(I),A(I),AZW
511 2(I),SHADE(I),UT(I),HI(I)
512 250 CONTINUE
513 WRITE (6,1690)
514 DO 260 I=1,NEXP
515 260 WRITE (6,1560) (SFAW(I,J),J=1,15)
516 IF (ASHRAE.EQ.1) GO TO 300
517 WRITE (6,1700)
518 WRITE (6,1710)
519 DO 270 I=1,NEXP
520 WRITE(6,1720)I,(G(I,J),J=1,NEXP)
521 270 CONTINUE
522 280 DO 290 I=1,NEXP

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523 DO 290 J=1,NEXP
524 290 G(I,J)=HR*G(I,J)
525 300 TIM=75.
526 IF (RUNTYP.EQ.2) CALL WINTER (A,UW,ITYPE,NEXP,CFMWT,DBIN,DBWT,UG,T
527 2GW,RHW,RHOW,UENDW,UCELNG,AENDW,ATCHT,AIRCHG)
528 QLITC=QLITY*AG*3.413*NOFLR
529 QEQPO=QEQPX*AG*3.413*NOFLR
530 DO 310 I=1,NEXP
531 QO(I)=0.
532 QI(I)=0.
533 310 CONTINUE
534 QRFC=0.
535 QRFI=0.
536 C DBM=TIM= REFERENCE TEMPERATURE
537 TA=TIM
538 MOT=0
539 TCLLD=0.
540 THTLD=0.
541 IF (IJKLMN.GT.1) GO TO 320
542 320 CONTINUE
543 NEND=DAYSKP+NDAY
544 IF (RUNTYP.NE.2) GO TO 340
545 NEND=7
546 DO 330 J=1,24
547 DB(J)=ZBLDG(4)-ZBLDG(5)*DBPF(J)
548 DPT(J)=DPMAX
549 WST(J)=0.
550 PBT(J)=29.921
551 TC(J)=0.
552 NTOC(J)=0
553 330 CONTINUE
554 340 DO 1250 ND=1,NEND
555 NSKP=ND-DAYSKP
556 IF (RUNTYP.EQ.2) GO TO 380
557 READ (7) DB,DPT,WBT,WST,PBT,TC,TOC,PR,PS,YEAR,MONTH,LDAY,ICITY
558 DO 350 IZ1=1,24
559 NTOC(IZ1)=TOC(IZ1)
560 350 CONTINUE
561 N=ND-DAYSKP
562 IF (N.LT.1) GO TO 1250
563 INDAY=DAYSKP+N
564 IF (IDETAL.EQ.0) GO TO 360
565 WRITE (6,1620) N,INDAY,YEAR,MONTH,LDAY
566 WRITE (6,1610) NAMERM
567 360 CONTINUE
568 KDAY=WKDAY(YEAR,MONTH,LDAY)
569 CALL HCLDAY (YEAR,MONTH,LDAY,KDAY,IHOL)
570 CALL DST (YEAR,MONTH,LDAY,DSTX,DSTY)
571 IDST=1
572 IF (MONTH.LT.4) IDST=0
573 IF (MONTH.GT.10) IDST=0
574 IF (MONTH.NE.4.OR.MONTH.NE.10) GO TO 370
575 IF (MONTH.EQ.4.AND.LDAY.LT.DSTX) IDST=0
576 IF (MONTH.EQ.10.AND.LDAY.GT.DSTY) IDST=0
577 370 DSX=IDST
578 JJ=1
579 IF (KDAY.EQ.7.OR.KDAY.EQ.1) JJ=2
580 IF (IHOL.EQ.1) JJ=2

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581 380 IF (RUNTYP.EQ.2) JJ=1
582 DO 390 J=1,24
583 QLITE(J)=QLITO*QLITX(J,JJ)
584 QEQUP(J)=QEQPO*QEQUX(J,JJ)
585 390 CONTINUE
586 IF (MONTH.EQ.MOT) GO TO 550
587 TG=TWG
588 IF (MONTH.GT.5.AND.MONTH.LT.10) TG=TGS
589 MCT=MONTH
590 S(4)=IEDAY(MCNTH)
591 IF (RUNTYP.EQ.2) S(4)=ZBLDG(3)
592 S(6)=IDST
593 IF (RLNTYP.EQ.2) S(6)=0.
594 S(7)=0.2
595 S(8)=1.0
596 S(33)=1.
597 IF (IDETAL.EQ.0) GO TO 400
598 WRITE (6,1730)
599 400 CONTINUE
600 DO 530 I=1,NEXP
601 IF (ITYPE(I).LT.5) GO TO 420
602 DO 410 J=1,24
603 QSUN(I,J)=0.
604 QGLAS(I,J)=0.
605 410 QSKY(I,J)=0.
606 GO TO 520
607 420 WAZ=AZW(I)
608 S(9)=WAZ
609 S(10)=90.
610 IF (ITYPE(I).EQ.1) S(10)=0.
611 SHDX(1)=SHAW(I,1)
612 SHDX(2)=SHAW(I,2)
613 SHDX(3)=SHAW(I,3)
614 SHDX(4)=SHAW(I,4)
615 SHDX(5)=SHAW(I,5)
616 SHDX(6)=SHAW(I,6)
617 SHDX(7)=SHAW(I,7)
618 SHDX(8)=SHAW(I,8)
619 SHDX(9)=SHAW(I,9)
620 SHDX(10)=SHAW(I,10)
621 SHDX(11)=SHAW(I,11)
622 SHDX(12)=SHAW(I,12)
623 SHDX(13)=SHAW(I,13)
624 SHDX(14)=SHAW(I,14)
625 SHDX(15)=SHAW(I,15)
626 430 CONTINUE
627 DO 510 J=1,24
628 QSKY(I,J)=0.
629 TIME=J
630 S(5)=TIME
631 CALL SUN
632 SALT(J)=S(20)
633 IF (S(25).GT.0.) GO TO 440
634 QSUN(I,J)=0.
635 QGLAS(I,J)=0.
636 GO TO 510
637 440 QSUN(I,J)=S(25)*AESP(I)
638 QGLAS(I,J)=0.

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639      PHI=S(21)*PI/180.
640      XQ=S(20)*PI/180.
641      CGSZ=SIN(XQ)
642      IF (SHD(I)) 460,460,450
643 450    SHDF(I,J)=0.
644      GO TO 480
645 460    SHDF(I,J)=1.
646      IF (SHDX(I)) 480,480,470
647 470    SHDX(16)=S(9)*PI/180.
648      CALL SHADOW (SHDX,PHI,COSZ,SHDF(I,J))
649 480    CONTINUE
650      IF (ITYPE(I).NE.3) GO TO 500
651      IF (IEXSTD.EQ.0) GO TO 490
652      IF (MONTH.GE.IEXMS.AND.MONTH.LE.IEXME) SHDF(I,J)=0.
653 490    CONTINUE
654      CALL GLASS (SHDF(I,J),SHADE(I),1.,1.,QGLAS(I,J))
655 500    CONTINUE
656      S34=S(25)-S(26)-S(27)
657      QSUN(I,J)=(S34+SHDF(I,J)+S(26)+S(27))*ABSP(I)
658 510    CONTINUE
659 520    IF (IDETAL.NE.0) WRITE (6,1660) I
660      IF (IDETAL.NE.0) WRITE (6,1670) (QSUN(I,J),J=1,24)
661      IF (IDETAL.NE.0) WRITE (6,1670) (QGLAS(I,J),J=1,24)
662 530    CONTINUE
663      DO 540 I=1,NEXP
664      DO 540 J=1,24
665      PGLAS(I,J)=QGLAS(I,J)
666 540    PSLN(I,J)=QSUN(I,J)
667 550    CONTINUE
668      IF (ND.NE.1) GO TO 640
669      DO 560 J=1,24
670      DO 560 I=1,NEXP
671 560    TOS(I,J)=DB(24-J+1)-TIM
672      DO 570 J=25,48
673      DO 570 I=1,NEXP
674 570    TOS(I,J)=TOS(I,J-24)
675      DO 580 I=1,NEXP
676      DO 580 J=1,48
677 580    TIS(I,J)=0.
678      TA=TIM
679      DO 590 J=1,48
680      TNEW(J)=0.
681 590    TATTIC(J)=0.
682      IF (ASHRAE) 640,640,600
683 600    DO 620 I=1,NEXP
684      DO 610 J=1,24
685 610    TIS(I,J)=RMDBS(24-J+1)-TIM
686      DO 620 J=25,48
687 620    TIS(I,J)=TIS(I,J-24)
688      DO 630 I=1,2
689      HEATG(I)=0.
690      HEATX(I)=0.
691      HEATIS(I)=0.
692      HLCG(I)=0
693      HLCX(I)=0.
694 630    HLCIS(I)=0.
695 640    CONTINUE
696 C      END OF INITIALIZATION

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697 C      TIME CALCULATION BEGINS HERE
698      DO 1150 NK=1,24
699      LL1=NK.GE.ISTART.AND.NK.LE.ILEAVE
700      LL2=NK.LT.ISTART.OR.NK.GT.ILEAVE
701      LL3=NVENT.NE.0.AND.DB(NK).LT.TV.AND.QL.LT.10.
702      IF (ITK.NE.0) GO TO 650
703      IF (ITHST.NE.1) GC TO 650
704      CALL TEMPSH (MONTH,JJ,NK,RMDBS,RMDBW,RMDBWO,RMDBSO,TA)
705 650    CONTINUE
706      IF (RUNTYP.NE.2) GC TO 660
707      FOT=4.
708      ACHG=ZROOM(9)
709      CM=1.
710      GO TO 670
711 660    WSTX=WST(NK)
712      CALL FC (WSTX,3,FOC,FOT,0)
713 C      AIR CHANGE AS A FUNCTION OF WIND SPEED
714 C      COBLENZ AND ACHENBACH 1963 ASHRAE TRANSACTION
715      WSTZ=WST(NK)*1.151
716      ACH=0.15+0.013*WSTZ+0.005*ABS(DB(NK)-TA)
717      ACHG=ACH*CONST.
718      CM=CCM(SALT(NK),NTDC(NK),TC(NK))
719 670    CFML=A(1)*ACHG/HT/60.+CFMIN
720      CFMLX=CFML
721      IF (LL1) GO TO 680
722      CFMV=0.
723      GO TO 690
724 680    IF (JJ.GT.1) CFMV=0.
725 690    CONTINUE
726      DO 720 I=1,NEXP
727      NRR=NR(I)
728      QSUN(I,NK)=PSUN(I,NK)*CM
729      QGLAS(I,NK)=PGLAS(I,NK)*CM
730      QSKY(I,NK)=0.
731      IF (ITYPE(I).EQ.1) QSKY(I,NK)=2.*(10.-TC(NK))
732      IF (NRR.LT.2) GO TO 720
733      DO 700 NTT=2,NRR
734 700    TOY(NTT)=TOS(I,NTT-1)
735      DO 710 NTT=2,NRR
736 710    TQS(I,NTT)=TOY(NTT)
737 720    CONTINUE
738      DO 800 I=1,NEXP
739      NRR=NR(I)
740      IF (ASHRAE.GT.0) TIS(I,1)=TA-TIM
741      K=IRF(I)
742      DO 730 J=1,NRR
743      XDUM(J)=X(K,J)
744      YDUM(J)=Y(K,J)
745      ZDUM(J)=Z(K,J)
746      TDUM(J)=TQS(I,J)
747      IF (ITYPE(I).EQ.6.OR.ITYPE(I).EQ.7) TDUM(J)=TIS(I,J)
748      IF (ITYPE(I).EQ.5) TDUM(J)=TG-TIM
749      T(I,J)=TIS(I,J)
750 730    CONTINUE
751      UX=U(I)
752      IF (H(I)) 750,750,740
753 740    H(I)=FOT
754      RX=1./UT(I)+1./(HI(I)+PR)

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755 RXX=RX+1./H(I)
756 U(I)=1./RXX
757 UX=1./RX
758 750 CONTINUE
759 IF (ITYPE(I).EQ.1.AND.UENDW.NE.0.) GO TO 760
760 GO TO 780
761 760 ATCACG=AIRCT+G*AIRNT
762 IF (LL1) ATCACG=AIRCT+G
763 CALL ATTIC (XDUM,YDUM,ZDUM,CR(I),NRR,UX,H(I),DB(NK),QSUN(I,NK),QSK
764 2Y(I,NK),TDUM,TATTIC,TNEW,TA,TIM,QRF0,QRF1,QO(I),QI(I),UENDW,UCELN
765 3G,AENDW,A(I),ATCMT,ATCACG)
766 DO 770 J=1,NRR
767 TNEW(J)=TDUM(J)
768 770 TDS(I,J)=TATTIC(J)
769 GC TO 800
770 780 CONTINUE
771 IF (RUNTYP.EQ.2) ITEMP=0
772 CALL OUTSID (XDUM,YDUM,ZDUM,CR(I),UX,H(I),DB(NK),TIM,QO(I),QI(I),Q
773 2SUN(I,NK),QSKY(I,NK),TDUM,TI,TNEW,TA,NRR)
774 DO 790 J=1,NRR
775 790 TDS(I,J)=TDUM(J)
776 800 CONTINUE
777 QDCPS(NK)=QDCUP(NK,JJ)*10.*(100.-TA)*QCU
778 QDCPL=10.*(TA-60.)*QDCUP(NK,JJ)*QCU
779 IF (TA-100.) 820,810,810
780 810 QDCPS(NK)=0.
781 QDCPL=400.*QDCUP(NK,JJ)*QCU
782 GO TO 840
783 820 IF (TA-65.) 830,840,840
784 830 QDCPS(NK)=350.*QDCUP(NK,JJ)*QCU
785 QDCPL=50.*QDCUP(NK,JJ)*QCU
786 840 DO 870 I=1,NEXP
787 NRR=NR(I)
788 IF (NRR.LT.2) GO TO 870
789 DO 850 NTT=2,NRR
790 850 TOY(NTT)=TIS(I,NTT-1)
791 DO 860 NTT=2,NRR
792 860 TIS(I,NTT)=TCY(NTT)
793 870 CONTINUE
794 IF (ASHRAE) 900,900,880
795 880 QSLMG=0.
796 QSUMX=0.
797 DO 890 I=1,NEXP
798 IF (ITYPE(I).LE.3.OR.ITYPE(I).EQ.8) QSUMX=QSUMX-QI(I)*A(I)
799 IF (ITYPE(I).EQ.3) QSUMG=QSUMG+GLAS(I,NK)*A(I)
800 890 CONTINUE
801 HEATG(1)=HEATG(2)
802 HEATG(2)=QSUMG
803 HEATX(1)=HEATX(2)
804 HEATX(2)=QSUMX+QDCPS(NK)+QDCUP(NK)
805 HEATIS(1)=HEATIS(2)
806 NKK=NK-1
807 IF (NKK.EQ.0) NKK=24
808 HEATIS(2)=QLITE(NKK)
809 HLCG(1)=HLCG(2)
810 HLCX(1)=HLCX(2)
811 HLCIS(1)=HLCIS(2)
812 ISC=1

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813 CALL RMRT (HEATG,HLCG,HEATX,HLCX,HEATIS,HLCIS,IW,IL,FC,ISC)
814 QL=HLCG(2)+HLCX(2)+HLCIS(2)+1.08*CFML*(DB(NK)-TA)
815 QGAIN=HEATG(2)+HEATX(2)+HEATIS(2)
816 QL=-QL
817 GO TO 965
818 900 CONTINUE
819 DO 960 I=1,NEXP
820 HI(I)=0.542
821 HTEST=TIS(I,1)
822 IF (I.NE.1) GO TO 930
823 IF (HTEST) 910,910,920
824 910 HI(I)=0.712
825 GO TO 960
826 920 HI(I)=0.162
827 GO TO 960
828 930 IF (I.NE.NEXP) GO TO 960
829 IF (HTEST) 940,940,950
830 940 HI(NEXP)=0.162
831 GO TO 960
832 950 HI(NEXP)=0.712
833 960 CONTINUE
834 965 CONTINUE
835 IF (NTVNT.EQ.0) GO TO 1010
836 IF (DB(NK).LT.DBVMIN.OR.DB(NK).GT.DBVMAX) GO TO 1010
837 970 IF (TA-DB(NK)) 1010,1010,980
838 980 IF (JJ.GT.1) GO TO 990
839 IF (LL1) GO TO 1010
840 990 IF (QL+10) 1000,1010,1010
841 1000 CFML=CFMLX+CFMNT
842 1010 CONTINUE
843 V(3)=0.
844 V(2)=CFML
845 V(1)=DB(NK)
846 IF (LL2 .CR. .NOT. LL3 .OR. JJ .GT. 1) GO TO 1011
847 V(3)=CFMV
848 1011 CONTINUE
849 CFMLNC=CFML+V(3)
850 IF (ASHRAE.GT.0.) GO TO 1040
851 V(4)=FRAS
852 V(5)=FLCG
853 V(6)=TIP
854 V(7)=QCMAX
855 V(8)=GHVAX
856 IF (JJ.GT.1) GO TO 1020
857 IF (LL2) GO TO 1020
858 V(9)=TUL
859 V(10)=TLL
860 GO TO 1030
861 1020 V(9)=RMDBSO
862 V(10)=RMDBWO
863 1030 CONTINUE
864 V(11)=TA
865 V(12)=FRAS
866 V(13)=HR
867 V(14)=METHOD
868 CALL RMTMK (V,TIF,QL,TA,NEXP,NK,ITK)
869 IF (LL1 .AND. LL3 .AND. QL .GT. 0. .AND. DB(NK) .GT. 60.) QL=0.
870 1040 CALL PSY2 (DB(NK),DPT(NK),PBT(NK),WBT(NK),PVO,WA,HA,VA,RHA)

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871 IF (ABS(QL).LT.10.)QL=0.
872 PLAT(NK)=-QOCPL*ZNORM
873 WV=WA
874 QOCPL=QOCPL/1060.
875 1080 WIN=(4.5*CFMLNC*WA+QOCPL)/4.5/CFMLNC
876 PVI=PB*WIN/(0.622+WIN)
877 RHIN=100.*PVI/PVSF(TA)
878 IF (RHIN.GT.100)RHIN=100.
879 IF (QL.EC.0.)GO TO 1086
880 IF (QL.GT.0.)GO TO 1085
881 IF (RHIN.GT.RHS)RHIN=RHS
882 1085 IF (QL.LT.0.)GO TO 1086
883 IF (RHIN.LT.RHW)RHIN=RHW
884 1086 CONTINUE
885 CALDB(NK)=TA
886 CALRH(NK)=RHIN
887 CALL CBRH (TA,RHIN,WIN)
888 AIRLAT(NK)=4.5*CFMLNC*(WIN-WA)*1060.*ZNORM
889 RALD(NK)=QLITE(NK)*FLCG*ZNORM
890 BASEL(NK)=(QLITE(NK)+QEQU(NK))*ZNORM
891 AIRLK(NK)=1.08*CFMLNC*(TA-DB(NK))*ZNORM
892 QSOL(NK)=PSUN(1,NK)
893 QLATNT=(4.5*CFMLNC*(WIN-WA)-QOCPL)*1060.
894 IF ((QL.GT.0..AND.RHIN.GT.RHW).OR.(QL.LT.0..AND.RHIN.LT.RHS))
895 1QLATNT=0.
896 IF (RUNTYP.EQ.2) GO TO 1100
897 IF (ASHRAE.EQ.0) GO TO 1100
898 CALL ADJUST (QL,QLATNT,MONTH,NK,JJ)
899 1100 CONTINUE
900 C
901 QLS(NK)=QL*ZNORM
902 QLL(NK)=QLATNT*ZNORM
903 IF (ABS(QLS(NK))-1.) 1110,1110,1120
904 1110 QLL(NK)=0.
905 1120 CONTINUE
906 IF (UENDW) 1150,1150,1130
907 1130 NRR=NR(1)
908 DO 1140 J=1,NRR
909 1140 TOS(1,J)=TNEW(J)
910 1150 CONTINUE
911 IF (RUNTYP.EQ.2.AND.ND.LT.7) GO TO 1250
912 QLMAX=ABS(QLS(1))
913 NMAX=1
914 TSUM=0.
915 QLDSUM=0.
916 CLDAY=0.
917 HLDAY=0.
918 DO 1200 NK=1,24
919 IF (QLMAX-ABS(QLS(NK))) 1160,1170,1170
920 1160 QLMAX=ABS(QLS(NK))
921 NMAX=NK
922 GO TO 1170
923 1170 CONTINUE
924 TSUM=TSUM+DB(NK)
925 QLDSUM=CLDSUM+QLS(NK)+QLL(NK)
926 QLDS=QLS(NK)+QLL(NK)
927 IF (QLDS) 1180,1180,1190
928 1180 CLDAY=CLDAY+QLDS

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929 GO TO 1200
930 1190 HLDAY=HLDAY+QLDS
931 1200 CONTINUE
932 TCLLD=TCLLD+CLDAY
933 THTLD=THTLD+HLDAY
934 DBA=TSUM/24.
935 QLMAX=QLS(NMAX)+QLL(NMAX)
936 IF (RUNTYP.EQ.2) N=1
937 IF (N.GT.1) GO TO 1210
938 WRITE (6,1520) NAMERM,MONTH
939 1210 CONTINUE
940 WRITE (6,1530) MONTH,LDAY,NMAX,QLMAX,CLDAY,HLDAY,DBA
941 IF (MOD(YEAR,4).EQ.0)MDAYS(2)=29
942 TMTHH=TMTHH+HLDAY
943 TMTHC=TMTHC+CLDAY
944 IF (QLMAX.LT.YMAXH)GO TO 1207
945 YMAXH=QLMAX
946 NMAXH=NMAX
947 LDAYH=LDAY
948 LMH=MONTH
949 1207 IF (QLMAX.GT.YMAXC)GO TO 1208
950 YMAXC=QLMAX
951 NMAXC=NMAX
952 LDAYC=LDAY
953 LMC=MONTH
954 1208 IF (LDAY.NE.MDAYS(MONTH))GO TO 1209
955 WRITE(6,1271)TMT+C,TMTHH
956 TMTHC=0.
957 TMTFH=0.
958 IF (ND.EC.NEND)GO TO 1209
959 WRITE(6,1272)
960 1209 CONTINUE
961 IF (ND.EQ.NEND) WRITE (6,1510)
962 IF (RUNTYP.NE.2) GO TO 1230
963 IF (ND.NE.NEND) WRITE (6,1510)
964 WRITE (6,1540) YEAR,MONTH,LDAY
965 WRITE (6,1490)
966 DO 1220 J=1,24
967 1220 WRITE (6,1500) J,CB(J),WBT(J),CALDB(J),CALRH(J),QLS(J),QLL(J)
968 WRITE (6,1510)
969 1230 CONTINUE
970 IF (IDETAL.EQ.0) GO TO 1240
971 WRITE (6,1750) DBA,QLDSUM
972 WRITE (6,1760) CLDAY,HLDAY
973 WRITE (6,1770) N,TCLLD,N,THTLD
974 1240 CONTINUE
975 IF (TAPE2.EQ.0) GO TO 1250
976 WRITE (TAPE2) NAMERM,MONTH,LDAY,DB,DPT,WBT,WST,PBT,TC,NTOC,CALDB,C
977 2ALRH,QLS,QLL,DBA,CLDAY,HLDAY,TCLLD,THTLD,QLITE,QEQU,QSOL,QOCPS,AI
978 3RLK
979 WRITE (10) QLS,PLAT,AIRLAT,DB,DPT,CALDB,RALD,BA SEL
980 1250 CONTINUE
981 CLDSUM=CLDSUM+TCLLD
982 HLDSUM=HLDSUM+THTLD
983 WRITE(6,1273)YMAXC,LMC,LDAYC,NMAXC,YMAXH,LMH,LDAYH,NMAXH
984 WRITE (6,1780) IJKLMN,CLDSUM,IJKLMN,HLDSUM
985 REWIND 7
986 1260 CONTINUE

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987      END FILE TAPE2
988      END FILE 10
989      STOP
990 C
991 C
992 1271 FORMAT(/// MONTHLY COOLING LOAD='E15.8,' BTU',/// MONTHLY HEATING
993 * LGAD='E15.8,' BTU')
994 1272 FORMAT(1H1,'MONTH DAY',7X,'MHR QLMAX',5X,'CLDAY',5X,'HLDAY',7X,
995 **DEA')
996 1273 FORMAT(' MAX CCCLING LOAD ='F10.0,' MONTH ='I3,' DAY ='I3,' HOU
997 *R ='I3,/' MAX HEATING LOAD ='F10.0,' MCNTH ='I3,' DAY ='I3,' H
998 *OUR ='I3')
999 1270 FORMAT (//34H RUNID,RUNTYP,ASHRAE,IDETAL,METHOD/41H RUNID.....
1000 2IDENTIFICATION OF THE RUN /42H 1 NEED RESPONSE FACTO
1001 3R DATA/42H 2 SKIP RESPONSE FACTOR DATA/26H RUNTYP...
1002 4.....TYPE OF RUN/56H 1 ENERGY CALCULATION ..NEEDS WE
1003 5ATHER TAPE/40H 2 DESIGN LOAD CALCULATION/52H
1004 6 3 DESIGN AND ENERGY LOAD CALCULATIONS/28H ASHRAE.....0
1005 7 USE RMTMP/47H 1 USE ASHRAE WEIGHTING FACTORS/37H
1006 8IDETAL.....0 NO DETAILED OUTPUT/34H 1 DETAILE
1007 9D OUTPUT/49H MET+CD.....0 REGULAR TREATMENT FOR THE ROOM/48H.
1008 *.....1 SPECIAL TREATMENT OF THE ROOM)
1009 1280 FORMAT (31H LIGHTING SCHEDULE FOR WEEKDAYS)
1010 1290 FORMAT (30H LIGHTING SCHEDULE FOR WEEKEND)
1011 1300 FORMAT (42H LIGHTING SCHEDULE FOR THE VACATION PERIOD)
1012 1310 FORMAT (38H EQUIPMENT USAGE SCHEDULE FOR WEEKDAYS)
1013 1320 FORMAT (32H EQUIPMENT SCHEDULE FOR WEEKENDS)
1014 1330 FORMAT (49H EQUIPMENT USAGE SCHEDULE FOR THE VACATION PERIOD)
1015 1340 FCRMAT (32H OCCUPANCY SCHEDULE FOR WEEKDAYS)
1016 1350 FORMAT (31H OCCUPANCY SCHEDULE FOR WEEKEND)
1017 1360 FORMAT (43H OCCUPANCY SCHEDULE FOR THE VACATION PERIOD)
1018 1370 FORMAT (42H THERMCSTAT SETTING FOR THE COOLING SEASON)
1019 1380 FORMAT (42H THERMOSTAT SETTING FOR THE HEATING SEASON)
1020 1390 FORMAT (22H RMDWVC,RMDBSO,RHW,RHS)
1021 1400 FORMAT (33H DATA SHEET NO 1:NDAY,NSKIP,TAPE2)
1022 1410 FORMAT (56H DATA SHEET NO 2 +3 :MONTH,DAY,ELAPS,DBMAX,RANGE,WBMAX,
1023 2 DBMWT,TGS,TGW,UG,LONG,LAT,TZN,ZLF,RHOW)
1024 1420 FORMAT (34H DATA SHEET NO 4: NAME OF THE ROOM)
1025 1430 FORMAT (35H DATA SHEET NO 5:IROT,ISKIP,INCLUDE)
1026 1440 FORMAT (85H DATA SHEET NO 8: ROOMNO,QLITY,GEOPY,QCU,FLCG,FRAS,TS,C
1027 2FNV,ARCHGS,ARCHGW,ARCHGM,ZNORM)
1028 1450 FORMAT (53H DATA SHEET NO.9: IW,IL,TUL,TLL,QCMAX,QHMAX,ITHST,ITK)
1029 1460 FORMAT (1)
1030 1470 FORMAT (56H DATA SHEET NO 13: UENDW,UCELNG,AENDW,ATCHT,AIRCHG,AIRN
1031 2T)
1032 1480 FORMAT (49H DATA SHEET NO 14: IEXTSD,IEXMS,IEXME,NTVNT,NVENT//)
1033 1490 FORMAT (//69H TIME DBOUT WBOU DBIN RHIN
1034 2 GLS QLL)
1035 1500 FORMAT (I10.4F10.1,2F10.0)
1036 1510 FORMAT (///)
1037 1520 FORMAT (///14H ROOM NAME= 9A4,9H MONTH= I6/56H DAY
1038 2MHR QLMAX CLDAY HLDAY DBA)
1039 1530 FORMAT (I4,I6,I10,3F10.0,F10.1)
1040 1540 FORMAT (13H ***** YEAR =,I5,14H ***** MONTH =,I3,12H ***** DAY =,I
1041 23/)
1042 1550 FORMAT (8H ROOMNO,6X2HHT,6X2HAG,3X5HNOFLR,5X3HOCU,2X6HARCHGS,2X6H
1043 2ARCHGW)
1044 1560 FORMAT (15F8.1)

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1045 1570 FORMAT (5X3HLAT,4X4HLONG,5X3HTZN,3X5HZNORM)
1046 1580 FORMAT (3X5HQLITY,3X5HQEQPX,4X4HCFMV,4X4HDBIN,6X2HTG,6X2HTV,4X4HRH
1047 2IN)
1048 1590 FORMAT (/31H UENCW UCELNG AENDW ATCHT)
1049 1600 FORMAT (6X,4HNEXP,7X,3HITK,5X,5HITHST/3(8X,I2))
1050 1610 FORMAT (1H ,6A6)
1051 1620 FORMAT (24H CLIMATIC DATA FOR DAY=,I5/27H DAYS ELAPSED SINCE JAN
1052 2 I=,I5,7H YEAR=,I5,8H MONTH=,I5,6H DAY=,I5)
1053 1630 FORMAT (1X,9A4)
1054 1640 FORMAT (9A4)
1055 1650 FCRMAT (I3,3 I10,8F10.2)
1056 1660 FORMAT (I10,F10.0)
1057 1670 FORMAT (24F5.0)
1058 1680 FORMAT (58H SURFACE NO ITYPE IHT IRF ABSP U
1059 2 H,9X,1HA9X,3HWAZ5X,5HSHADEBX,2FUT8X,2HHI)
1060 1690 FORMAT (///21H SHADOW CASTING DATA/121H HT FL FP
1061 2 AW BWL BWR D FP1 A1 B1 C1
1062 3 FP2 A2 B2 C2)
1063 1700 FORMAT (///33H RADIATION INTERCHANGE FACTORS)
1064 1710 FORMAT (108H SURFACE 1 2 3 4
1065 2 5 6 7 8 9 10)
1066 1720 FORMAT (I10,10F10.3)
1067 1730 FORMAT (27H SCLAR DATA (QSUN/QGLASS))
1068 1740 FORMAT (4H /5H )
1069 1750 FORMAT (6H DEA =,F6.2/9H QLDSUM =,F10.0//)
1070 1760 FORMAT (36H TOTAL CCCLING CONSUMPTION PER DAY =,F10.0,4H BTU/36H T
1071 2DOTAL HEATING CONSUMPTION PER DAY =,F10.0,4H BTU)
1072 1770 FORMAT (49H TOTAL COOLING CONSUMPTION FOR THE ROOM OVER THE ,I3,14
1073 2H DAY PERIOD =,E11.5,4H BTU/49H TOTAL HEATING CONSUMPTION FOR THE
1074 3 ROOM OVER THE ,I3,13H DAY PERIOD =,E11.5,4H BTU)
1075 1780 FORMAT (31H TOTAL COOLING CONSUMPTION FOR ,I2,8H ROOMS =,E11.5,4H
1076 2BTU/31H TOTAL HEATING CONSUMPTION FOR ,I2,8H ROOMS =,E11.5,4H BTU)
1077 1790 FORMAT (///39H CONGRATULATIONS## NOW YOU ARE ON NBSLD)
1078 1800 FCRMAT (/46H WE ASSUME YOU HAVE ALREADY PREPARED THE DATA)
1079 1810 FORMAT (52H ON NBS DATA FORMS..IF YOU HAD NOT ,PLEASE TURN OFF)
1080 1820 FORMAT (57H THE TERMINAL AND HAVE YOUR DATA READY ON THE DATA FOR
1081 2MS)
1082 C
1083 END
1084 @FOR,IS A2
1085 SUBROUTINE ABCD2 (Z,K,L,G,A,B,C,D,NL)
1086 C
1087 C *****
1088 C
1089 DIMENSION AX(10),BX(10),CX(10),DX(10),G(10)
1090 REAL K(10),L(10)
1091 PI=4.*ATAN(1.)
1092 PP=PI*0.5
1093 DO 50 I=1,NL
1094 IF (G(I)) 40,40,10
1095 10 IF (Z) 30,30,20
1096 20 ZG=SQRT(Z/G(I))
1097 ZGL=ZG*L(I)
1098 CG=SIN(ZGL)
1099 C1=COS(ZGL)
1100 S1=CG/ZGL
1101 S2=(S1-C1)/ZGL/ZGL
1102 AX(I)=C1

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1103      BX(I)=L(I)/K(I)*S1
1104      CX(I)=-ZQL*K(I)/L(I)*CO
1105      DX(I)=C1
1106      GO TO 50
1107 30    AX(I)=1.
1108      CX(I)=0.
1109      DX(I)=1.
1110      BX(I)=L(I)/K(I)
1111      GO TO 50
1112 40    AX(I)=1.
1113      BX(I)=1/K(I)
1114      CX(I)=0.
1115      DX(I)=1.
1116 50    CONTINUE
1117      A=AX(I)
1118      B=BX(I)
1119      C=CX(I)
1120      D=DX(I)
1121      IF (NL.LT.2) GO TO 60
1122      CALL MULT (AX,BX,CX,DX,A,B,C,D,NL)
1123 60    RETURN
1124      C
1125      END
1126 8FOR, IS A3
1127      SUBRCUTINE ABCDP2 (Z,K,L,G,AP,BP,CP,DP)
1128      C
1129      C *****
1130      C
1131      REAL K,L
1132      PI=4.*ATAN(1.)
1133      IF (G) 30,30,10
1134 10    PP=PI/4./G
1135      IF (Z) 40,40,20
1136 20    ZC=SQRT(Z/G)
1137      ZQL=ZQ*L
1138      X=L*L*0.5/G
1139      RES=L/K
1140      CO=SIN(ZQL)
1141      C1=COS(ZQL)
1142      S1=CO/ZQL
1143      S2=(S1-C1)/ZQL/ZQL
1144      AP=X*S1
1145      BP=X*RES*S2
1146      CP=X*(S1+C1)/RES
1147      DP=X*S1
1148      GO TO 50
1149      C
1150 30    AP=0.
1151      BP=0.
1152      CP=0.
1153      DP=0.
1154      GO TO 50
1155      C
1156 40    CONTINUE
1157      X=L*L*0.5/G
1158      AP=X
1159      BP=X*L/K/3
1160      CP=K/L*X*2.

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1161      DP=X
1162      GC TO 50
1163      C
1164 50    RETURN
1165      C
1166      END
1167 8FCR, IS A4
1168      SUBROUTINE ADJUST (QL,QLATNT,MONTH,NK,JJ)
1169      C
1170      C *****
1171      C
1172      IF (MCNTF.GE.6.AND.MCNTF.LE.9) GO TO 30
1173      IF (QL) 10,10,20
1174 10    IF (JJ.GT.1) GO TO 50
1175      IF (NK.LT.8.OR.NK.GT.17) GO TO 50
1176 20    RETURN
1177      C
1178 30    IF (QL) 40,50,50
1179 40    IF (JJ.GT.1) GO TO 50
1180      IF (NK.LT.8.OR.NK.GT.17) GO TO 50
1181      RETURN
1182      C
1183 50    QL=0.
1184      QLATNT=0.
1185      RETURN
1186      C
1187      END
1188 8FOR, IS A5
1189      SUBRCUTINE ATTIC (X,Y,Z,CR,NR,UX,FQ,DB,OSUN,OSKY,TDS,TI,TINEW,TA,
1190      2 TIM,ORFC,ORFI,OO,QI,UENDW,UCELNG,AENDW,ARCOF,ATCHT,ATCACG)
1191      C
1192      C *****
1193      C
1194      C THIS ROUTINE CALCULATES HEAT INPUT TO THE ROOM BELOW THE
1195      C ATTIC CEILING. IT ALSO CALCULATES ATTIC TEMPERATURE
1196      C X,Y,Z RESPONSE FACTORS FOR ROOF... INSIDE SURFACE THERMAL
1197      C RESISTANCE IS INCLUDED
1198      C CR COMMON RATIO OF THE ROOF RESPONSE FACTORS
1199      C NR NUMBER OF SIGNIFICANT RESPONSE FACTORS TO BE USED
1200      C UX ROOF OVER ALL HEAT CONDUCTANCE EXCLUDING THE EXTERIOR SURFACE
1201      C THERMAL RESISTANCE
1202      C FC ROOF EXTERIOR SURFACE THERMAL TRANSFER COEFFICIENT
1203      C DB OUTDOOR DRY-BULB TEMPERATURE
1204      C OSUN SOLAR RADIATION OVER THE ROOF
1205      C OSKY RADIATION TO THE SKY
1206      C TCS ROOF SURFACE TEMPERATURE HISTORY
1207      C TI ATTIC TEMPERATURE HISTORY
1208      C TQNEW NEW OUTSIDE SURFACE TEMPERATURE
1209      C TINEW NEW ATTIC TEMPERATURE
1210      C QRFO HEAT CONDUCTED INTO THE ROOF SURFACE
1211      C QRFI HEAT CONDUCTED INTO THE ATTIC FROM THE ROOF
1212      C QO,QI HEAT CONDUCTED INTO THE ROOM BELOW THE ATTIC
1213      C UENDW OVERALL HEAT TRANSFER COEFFICIENT OF THE END WALL
1214      C UCELNG OVERALL HEAT TRANSFER COEFFICIENT OF THE CEILING
1215      C BETWEEN THE ATTIC AIR AND ROOM AIR BELOW
1216      C AENDW AREA OF THE ATTIC END WALLS
1217      C ARCOF TOTAL AREA OF THE CEILING
1218      C ATCHT ATTIC HEIGHT

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1219 C ATCAGG ATTIC AIR CHANGE PER HOUR
1220 C ALL UNITS IN ENGLISH UNIT ALL LENGTH IN FT
1221 DIMENSION TOS(1),TI(1),X(1),Y(1),Z(1),TOY(50)
1222 CFM=ATCAGG*ATCHT*AROOF/60.
1223 BB=DB-TIM
1224 TAM=TA-TIM
1225 XNUM=QSLN-QSKY+FO*BB
1226 YNUM=UENDW*BB*AENDW+1.08*CFM*BB+UCELNG*TAM*AROOF
1227 YDEN=UENDW*AENDW+UCELNG*AROOF+1.08*CFM
1228 IF (NR.LT.2) GO TO 10
1229 GO TO 20
1230
1231 C
1232 10 TAM=TI(1)
1233 TCNEW=(XNUM+UX*TAM)/(UX+FO)
1234 QRFQ=UX*(TAM-TONEW)
1235 QRFI=CRFC
1236 TINEW=(YNUM+UX*ARCOF*TONEW)/(YDEN+UX*AROOF)
1237 TOS(1)=TCNEW
1238 QG=UCELNG*(TAM-TINEW)
1239 QI=CC
1240 TI(1)=TINEW
1241 RETURN
1242 C
1243 20 SUMX=X(1)*TI(1)
1244 SUMZ=0.
1245 SUMY=Y(1)*TI(1)
1246 SUMXY=0.
1247 DO 50 J=2,NR
1248 SUMY=SUMY+Y(J)*TI(J)
1249 SUMX=SUMX+X(J)*TI(J)
1250 SUMXY=SUMXY+Y(J)*TOS(J)
1251 SUMZ=SUMZ+Z(J)*TOS(J)
1252 XNUM=XNUM+SUMY-SUMZ+CR*QRFQ
1253 TONEW=XNUM/(Z(1)+FO)
1254 TOS(1)=TONEW
1255 SUMZ=SUMZ+Z(1)*TCS(1)
1256 SUMXY=SUMXY+Y(1)*TOS(1)
1257 QRFQ=SUMY-SUMZ+CR*QRFQ
1258 QRFI=SUMX-SUMXY+CR*QRFI
1259 DO 30 J=2,NR
1260 TDY(J)=TI(J-1)
1261 DO 40 J=2,NR
1262 TI(J)=TOY(J)
1263 SUMX=0.
1264 SUMY=SUMXY
1265 SUMXY=0.
1266 DO 60 J=2,NR
1267 SUMX=SUMX+X(J)*TI(J)
1268 SUMXY=SUMXY+Y(J)*TI(J)
1269 YNUM=YNUM-AROOF*(SUMX-SUMY+CR*QRFI)
1270 YDEN=AROOF*X(1)+YDEN
1271 TINEW=YNUM/YDEN
1272 SUMX=SUMX+TINEW*X(1)
1273 QRFI=SUMX-SUMY+CR*QRFI
1274 SUMXY=SUMXY+Y(1)*TINEW
1275 QRFQ=SUMXY-SUMZ+CR*QRFQ
1276 QG=UCELNG*(TAM-TINEW)
1277 QI=QG

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1277 TI(1)=TINEW
1278 RETURN
1279 C
1280 END
1281 @FOR,IS A6
1282 FUNCTION CCM (SALT,NTYPE,TC)
1283 C
1284 C *****
1285 C
1286 REAL CC1(10) /.60,.60,.58,.58,.57,.53,.49,.43,.35,.27/
1287 REAL CC2(10) /.88,.88,.88,.87,.85,.83,.79,.73,.61,.46/
1288 REAL CC3(10) /.84,.83,.83,.82,.80,.79,.74,.67,.60,.49/
1289 REAL CC4(10) /1.,1.,1.,1.,.99,.98,.95,.90,.84,.74/
1290 ITC=TC
1291 IF (ITC.NE.0) GO TO 10
1292 CCM=1.
1293 GO TO 60
1294 C
1295 10 CONTINUE
1296 IF (SALT-45.) 40,40,20
1297 20 IF (NTYPE.EQ.0) GO TO 30
1298 CCM=CC2(ITC)
1299 GO TO 60
1300 C
1301 30 CCM=CC4(ITC)
1302 GO TO 60
1303 C
1304 40 IF (NTYPE.EQ.0) GO TO 50
1305 CCM=CC1(ITC)
1306 GC TO 60
1307 C
1308 50 CCM=CC3(ITC)
1309 60 RETURN
1310 C
1311 END
1312 @FOR,IS A7
1313 SUBROUTINE DBRH (DB,RH,W)
1314 C
1315 C *****
1316 C
1317 PVS=PVSF (DB)
1318 PV=RH*PVS/100.
1319 W=0.622*PV/(29.92-PV)
1320 RETURN
1321 C
1322 END
1323 @FOR,IS A8
1324 SUBROUTINE DECODE (WPOSX,WLONGX,NUM,OUTPUT,MM,YR,MO,DAY,LOCAL)
1325 C
1326 C *****
1327 C
1328 C THIS SUBROUTINE PRODUCES HOURLY DATA OF UP TO 10 WEATHER
1329 C PARAMETERS FOR A GIVEN YEAR,MO AND DATE
1330 C TAPE FCSOTION FOR EACH OF TEN PARAMETERS ARE
1331 C PARAMETERS WPOSX WLONGX
1332 C WIND SPEED 13 3
1333 C WIND DIRECTION 11 2
1334 C DRY-BULB TEMP 16 3

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1335 C      WET-BULB TEMP      19      3
1336 C      DEW-POINT TEMP    22      3
1337 C      BARMETRIC PRESS   34      4
1338 C      TOTAL CLOUD AMOUNT 43      1
1339 C      OPAQUE CLOUD COVER 44      1
1340 C      PRECIPITATION(LIQUID) 68    2
1341 C      PRECIPITATION(FRZ) 70      3
1342 C      TAPE FCSITICN CN TAPE 280
1343 C      SOLAR DATA        14      4
1344 C      ELEVATION ANGLE    18      2
1345 C      TOTAL CLGUD        42      1
1346 C      1ST LAYER TYPE OF CLOUD 46  1
1347 C      YR      YEAR
1348 C      MO      MONTH
1349 C      DAY     DAY
1350      INTEGER IPS(24), ICHAR(2000), WPOS, WLONG, OUTPUT(24,10), YR, DAY, WORD,
1351      2 WORDX(20), WPOSX(10), WLONGX(10), TAPE1, TAPE2
1352      COMMON TAPE1, TAPE2, ND1, ND2, INPUT(1100)
1353      IAASS=1000000
1354      IF (MM.NE.0) GO TO 90
1355      DO 10 I=1,4
1356      CALL WD (0)
1357      DO 10 JJ=1,498
1358      KK=498*(I-1)+JJ
1359 10      ICHAR(KK)=INPUT(JJ)
1360      DO 20 I=1,15
1361      IW=ICHA(I)
1362      CALL WDX (IW)
1363 20      ICHAR(I)=IW
1364      YR=ICHA(10)*10+ICHA(11)+1900
1365      MO=ICHA(12)*10+ICHA(13)
1366      DAY=ICHA(14)*10+ICHA(15)
1367      LOCAL=ICHA(9)
1368      IPWR=1
1369      DO 30 I=1,4
1370      IPWR=IPWR*10
1371 30      LOCAL=LCCAL+ICHA(9-I)*IPWR
1372      DO 80 KU=1,NUM
1373      WPCS=WPOSX(KU)
1374      WLONG=WLONGX(KU)
1375      DO 40 I=1,6
1376      IPS(I)=15+WPOS+80*(I-1)
1377      DO 40 J=1,3
1378      II=I+J*6
1379 40      IPS(II)=IPS(I)+J*498
1380      DO 70 I=1,24
1381      KI=IPS(I)
1382      KL=KI+WLCNG-1
1383      DO 50 L2=KI,KL
1384      IW=ICHA(L2)
1385      CALL WDX (IW)
1386 50      ICHAR(L2)=IW
1387      LONG=WLCNG-1
1388      IF (ICHA(KI).EQ.IAASS.AND.WLONG.GT.1) LONG=WLONG-2
1389      WORD=AES(ICHA(KL))
1390      IF (LCNG.EC.0) GO TO 70
1391      IPWR=1
1392      DC 60 JK=1, LONG

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1393      IPWR=IPWR*10
1394 60      WORD=WORD+ICHA(KL-JK)*IPWR
1395      IF (ICHA(KL).LT.0) WORD=-WORD
1396 70      OUTPUT(I,KU)=WORD
1397 80      CONTINUE
1398      GO TO 240
1399 C
1400 90      CALL WD (1)
1401      JZ=0
1402      DO 100 J=1,991,66
1403      JX=J
1404      JZ=JZ+1
1405      IW=INPUT(J)
1406      CALL WDX (IW)
1407      IF (IW.NE.IAASS) GO TO 110
1408 100      CONTINUE
1409 110      IF (JX.LT.991) GO TO 130
1410      DO 120 KU=1,NUM
1411      DO 120 J=1,24
1412 120      OUTPUT(J,KU)=IAASS
1413      GO TO 240
1414 C
1415 130      JY=JX+20
1416      DO 140 I=JX, JY
1417      IW=INPUT(I)
1418      CALL WDX (IW)
1419 140      ICHAR(I)=IW
1420      YR=ICHA(JX+5)*10+ICHA(JX+6)+1900
1421      DAY=ICHA(JX+9)*10+ICHA(JX+10)
1422      MO=ICHA(JX+7)*10+ICHA(JX+8)
1423      IF (DAY.GT.0) GO TO 150
1424      IY=AES(DAY)
1425      IF (IY.LT.20) DAY=DAY+20
1426      IF (IY.GE.20) DAY=DAY+40
1427 150      CONTINUE
1428      LOCAL=ICHA(JX+4)
1429      IPWR=1
1430      DC 160 I=1,4
1431      IPWR=IPWR*10
1432      IF (ICHA(JX+4-I).GT.0) GO TO 160
1433      LOCAL=IAASS
1434      GO TO 170
1435 160      LOCAL=LOCAL+ICHA(JX+4-I)*IPWR
1436 170      CONTINUE
1437      IHR=3+JZ
1438      IF (WPCS.EQ.0) GO TO 240
1439      DO 230 KU=1,NUM
1440      WPCS=WPCSX(KU)
1441      WLONG=WLONGX(KU)
1442      DO 180 I=1,24
1443 180      OUTPUT(I,KU)=0
1444      DO 210 I=1,16
1445      KI=WPOS+66*(I-1)
1446      KL=KI+WLONG-1
1447      DO 190 L2=KI,KL
1448      IW=INPUT(L2)
1449      CALL WDX (IW)
1450 190      ICHAR(L2)=IW

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1451      LCNG=WLCNG-1
1452      IF (ICHAR(KI),EQ.IASS.AND.WLONG.GT.1) LONG=WLONG-2
1453      WORD=ICHAR(KL)
1454      IF (LCNG.EC.0) GO TO 210
1455      IPWR=1
1456      IPWR=IPWR*10
1457      DO 200 JK=1,LCNG
1458      IF (ICHAR(KL).LT.0) WORD=-WORD
1459 200    WORD=WORD+ICHAR(KL-JK)*IPWR
1460 210    WORDX(I)=WORD
1461      DO 220 I=1,16
1462      KK=I+3
1463 220    OUTPUT(KK,KU)=WORDX(I)
1464 230    CONTINUE
1465 240    RETLRN
1466      C
1467      END
1468 @FOR, IS A9
1469      SUBROUTINE DERVT (A,B,C,D,AP,BP,CP,DP,APP,BPP,CFP,DPP,N)
1470      C
1471      C *****
1472      C
1473      DIMENSION A(N),B(N),C(N),D(N),AP(N),BP(N),CP(N),DP(N),AT(10),
1474 2 BT(10),CT(10),DT(10),ATT(10),BTT(10),CTT(10),DTT(10)
1475      DO 30 I=1,N
1476      DO 20 J=1,N
1477      IF (I.EQ.J) GO TO 10
1478      AT(J)=A(J)
1479      BT(J)=B(J)
1480      CT(J)=C(J)
1481      DT(J)=D(J)
1482      GO TC 20
1483 10    AT(J)=AP(J)
1484      BT(J)=BP(J)
1485      CT(J)=CP(J)
1486      DT(J)=DP(J)
1487 20    CONTINUE
1488 30    CALL MULT (AT,BT,CT,DT,ATT(I),BTT(I),CTT(I),DTT(I),N)
1489      APP=ATT(I)
1490      BPP=BTT(I)
1491      CPP=CTT(I)
1492      DPP=DTT(I)
1493      DO 40 I=2,N
1494      AFF=APP+ATT(I)
1495      BPP=BPP+BTT(I)
1496      CFP=CPP+CTT(I)
1497 40    DPP=DPP+DTT(I)
1498      RETURN
1499      C
1500      END
1501 @FCR, IS A10
1502      SUBROUTINE DST (YR,MO,DAY,DSTX,DSTY)
1503      C
1504      C *****
1505      C
1506      INTEGER YR,DAY,DSTX,DSTY
1507      NDAY=WKDAY(YR,MO,DAY)
1508      IF (MO.LT.4.CR.MC.GT.10) GO TO 10

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1509      IF (MO.EQ.4.AND.DAY.LT.24) GO TO 10
1510      IF (NDAY.EQ.1) DSTX=DAY
1511      IF (MO.EQ.10.AND.DAY.LT.24) GO TO 10
1512      IF (NDAY.EQ.1) DSTY=DAY
1513 10    CONTINUE
1514      RETURN
1515      C
1516      END
1517 @FOR, IS A11
1518      FUNCTION DPF (PV)
1519      C
1520      C *****
1521      C
1522      C THIS SUBROUTINE CALCULATES DEW-POINT TEMPERATURE FOR GIVEN VAPOR P
1523      IF (PV) 10,10,20
1524 10    GO TO 40
1525      C
1526 20    CONTINUE
1527      Y=LOG(PV)
1528      IF (PV.GT.0.1836) GO TO 30
1529      DPF=71.98+24.273*Y+0.8927*Y*Y
1530      GO TO 40
1531      C
1532 30    DPF=79.047+30.575*Y+1.8893*Y*Y
1533 40    RETURN
1534      C
1535      END
1536 @FOR, IS A12
1537      SUBROUTINE ERROR (IDATA,K)
1538      C
1539      C *****
1540      C
1541      DIMENSION MAX(10) /100,100,150,150,150,3500,10,10,99,999/
1542      DIMENSION MIN(10) /0,0,-40,-40,-40,2000,0,0,0,0/ ,IDATA(24)
1543      DC 10 J=1,24
1544      IZ=IDATA(J)
1545      IF (IZ.GT.MAX(K)) GO TO 10
1546      IF (IZ.LT.MIN(K)) GC TC 10
1547      GO TO 20
1548 10    CONTINUE
1549 20    IDATA(1)=IZ
1550      DO 40 J=2,24
1551      IZ=IDATA(J)
1552      IF (IZ.GT.MAX(K)) GO TO 30
1553      IF (IZ.LT.MIN(K)) GC TO 30
1554      GO TO 40
1555 30    IDATA(J)=IDATA(J-1)
1556 40    CONTINUE
1557      RETURN
1558      C
1559      END
1560 @FOR, IS A13
1561      FUNCTION F (A,B,C)
1562      C
1563      C *****
1564      C
1565      C BC = RECEIVING SURFACE
1566      C AB = SENDING SURFACE

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1567 C F(A,B,C)= FRCH AB TO BC
1568 PI=3.14159
1569 C F(C,E,A)=F(A,B,C)*A/C
1570 C (A*B)*F(A,B,C)=(B*C)*F(C,B,A)
1571 X=A/B
1572 Y=C/B
1573 Z=X*X+Y*Y
1574 A2=LOG((1.+X*X)*(1.+Y*Y)/(1.+Z))
1575 A3=Y*Y*LOG(Y*Y*(1.+Z)/(1.+Y*Y)/Z)
1576 A4=X*X*LOG(X*X*(1.+Z)/(1.+X*X)/Z)
1577 A5=Y*ATAN(1./Y)
1578 A6=X*ATAN(1./X)
1579 A7=SQRT(Z)*ATAN(1./SQRT(Z))
1580 SUM=(A2+A3+A4)/4.+A5+A6-A7
1581 F=SUM/PI/Y
1582 RETURN
1583 C
1584 END
1585 @FCR, IS A14
1586 SUBROUTINE FCTR (L,W,H,SF)
1587 C
1588 C *****
1589 C
1590 REAL L,SF(6,6)
1591 C THIS ROUTINE CALCULATES BASIC RADIATION SHAPE FACTORS FOR A ROOM.
1592 C RADIATION SHAPE FACTOR F(H,L,W)=(H*L)---*(L*W)
1593 C TO FROM C S W N E
1594 C C CEILING 0 FHLW FHWL FHLW FHWL
1595 C S SOUTH WALL FWLH 0 FWHL RMS FWHL
1596 C W WEST WALL FLWH FLHW 0 FLHW RMW
1597 C N NORTH WALL FWLH RMN FWHL 0 FWHL
1598 C E EAST WALL FLWH FLHW RME FLHW 0
1599 C F FLOOR RMF FHLW FHWL FHLW FHWL
1600 C
1601 C RM = REMAINDER
1602 FHLW=F(H,L,W)
1603 FHWL=F(H,W,L)
1604 FLWH=F(W,L,H)
1605 FWHL=F(W,H,L)
1606 FLHW=F(L,W,H)
1607 FLHW=F(L,H,W)
1608 RMC=1.-2.*(FHLW+FHWL)
1609 RMS=1.-2.*(FLWH+FHWL)
1610 RMW=1.-2.*(FLWH+FLHW)
1611 RMN=RMS
1612 RME=RMW
1613 RMF=RMC
1614 SF(1,1)=0.
1615 SF(1,2)=FHLW
1616 SF(1,3)=FHWL
1617 SF(1,4)=FHLW
1618 SF(1,5)=FHWL
1619 SF(1,6)=RMC
1620 SF(2,1)=FHLW
1621 SF(2,2)=0.
1622 SF(2,3)=FHWL
1623 SF(2,4)=RMS
1624 SF(2,5)=FHWL

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1625 SF(2,6)=FHLW
1626 SF(3,1)=FLWH
1627 SF(3,2)=FLHW
1628 SF(3,3)=0.
1629 SF(3,4)=FLHW
1630 SF(3,5)=RMW
1631 SF(3,6)=FLWH
1632 SF(4,1)=FHLW
1633 SF(4,2)=RMN
1634 SF(4,3)=FHWL
1635 SF(4,4)=0.
1636 SF(4,5)=FHWL
1637 SF(4,6)=FHLW
1638 SF(5,1)=FLWH
1639 SF(5,2)=FLHW
1640 SF(5,3)=RMF
1641 SF(5,4)=FLHW
1642 SF(5,5)=0.
1643 SF(5,6)=FLWH
1644 SF(6,1)=RMF
1645 SF(6,2)=FHLW
1646 SF(6,3)=FHWL
1647 SF(6,4)=FHLW
1648 SF(6,5)=FHWL
1649 SF(6,6)=0.
1650 RETURN
1651 C
1652 END
1653 @FCR, IS A15
1654 SUBROUTINE FO (V,IS,FOC,FOT,IWD)
1655 C
1656 C *****
1657 C THIS SUBROUTINE CALCULATES OUTSIDE SURFACE HEAT TRANSFER
1658 C COEFFICIENTS,FOT AND FOC
1659 C FOT....RADIATION PLUS CONVECTION
1660 C FOC....CONVECTION
1661 C V.....WIND VELOCITY IN KNOTS
1662 C
1663 C
1664 C *****
1665 C
1666 C DIMENSION A(6) /0.,0.001,0.,-0.002,0.,-0.00125/ ,B(6) /.464,0.320,
1667 C 2 0.330,0.315,0.244,0.262/ ,C(6) /2.04,2.20,1.90,1.45,1.80,1.45/
1668 C
1669 C VP=V*1.153
1670 C FOT=A(IS)*VP*VP+B(IS)*VP+C(IS)
1671 C IWD=1 IF THE SURFACE IS WINDWARD OR PARALLEL TO THE WIND
1672 C IWD=0 IF THE SURFACE IS LEEWARD
1673 C IF (IWD.EQ.0) GO TO 20
1674 C IF (VP-7.0) 20,20,10
1675 C 10 FOC=0.23*VP+1.02
1676 C GO TO 30
1677 C
1678 C 20 FOC=2.63
1679 C 30 RETURN
1680 C
1681 C END
1682 @FOR, IS A16

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1683 SUBROUTINE GLASS (SHADCW,SHDCF,GLTYP,GLAZE,SHGF)
1684 C
1685 C *****
1686 C
1687 DIMENSICN TR(9),SH(25)
1688 REAL LAT,LCNG
1689 COMMON /SCL/ LAT,LCNG,TZN,WAZ,WT,CN,DST,LPYR,S(35)
1690 TR(7)=S(19)
1691 TR(8)=GLTYP
1692 TR(9)=GLAZE
1693 CALL TAR (TR)
1694 SH(1)=S(24)
1695 SH(2)=S(22)
1696 SH(3)=S(23)
1697 SH(4)=S(19)
1698 SH(5)=0.5
1699 SH(6)=0.5
1700 SH(7)=0.25
1701 SH(8)=0.
1702 SH(9)=0.7
1703 SH(10)=SHADDW
1704 SH(11)=SHDCF
1705 SH(12)=TR(1)
1706 SH(13)=TR(2)
1707 SH(14)=TR(3)
1708 SH(15)=TR(5)
1709 SH(16)=TR(4)
1710 SH(17)=TR(6)
1711 CALL SHG (SH)
1712 SHGF=SH(18)
1713 RETURN
1714 C
1715 END
1716 @FOR,IS A17
1717 SUBROUTINE GPF (U,ZL,Z)
1718 C
1719 C *****
1720 C
1721 DIMENSION Z(1)
1722 PI=4.*ATAN(1.)
1723 SQTP=SQRT(PI)
1724 PI2=2./PI
1725 EB=0.001
1726 DB=0.1
1727 WRITE (6,30)
1728 WRITE (6,40)
1729 Z(1)=2*ZL*SQRT(U)/SQTP
1730 ZZ=Z(1)
1731 Z(2)=Z(1)*(SQRT(2.)-2.)
1732 DO 10 K=3,50
1733 ZK=K
1734 10 Z(K)=Z(1)*(SQRT(ZK)-2.*SQRT(ZK-1)+SQRT(ZK-2.))
1735 DO 20 K=1,50
1736 20 WRITE (6,50) K,Z(K)
1737 RETURN
1738 C
1739 C
1740 30 FORMAT (50H0 RESPONSE FACTORS FOR SEMI-INFINITE BED

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1741 40 FORMAT (50H0 K Z(K)
1742 50 FORMAT (11I0,3F10.5)
1743 C
1744 END
1745 @FCR,IS A18
1746 SLBRCUTINE HOLIDAY (YR,MC,DAY,NDAY,HOL)
1747 C
1748 C *****
1749 C
1750 INTEGER YR,DAY,HOL,WKDAY
1751 NDAY=WKDAY(YR,MC,DAY)
1752 HOL=0
1753 IF (MO.EQ.1.AND.DAY.EQ.1) HOL=1
1754 IF (MO.EQ.12.AND.DAY.EQ.31.AND.NDAY.EQ.6) HOL=1
1755 IF (MO.EQ.1.AND.CAY.EQ.2.AND.NDAY.EQ.2) HOL=1
1756 IF (MO.EQ.2.AND.CAY.EQ.22) HOL=1
1757 IF (MO.EQ.2.AND.DAY.EQ.21.AND.NDAY.EQ.6) HOL=1
1758 IF (MO.EQ.2.AND.CAY.EQ.23.AND.NDAY.EQ.2) HOL=1
1759 IF (MO.EQ.5.AND.CAY.EQ.30) HOL=1
1760 IF (MO.EQ.5.AND.DAY.EQ.29.AND.NDAY.EQ.6) HOL=1
1761 IF (MO.EQ.5.AND.DAY.EQ.31.AND.NDAY.EQ.2) HOL=1
1762 IF (MC.EQ.7.AND.CAY.EQ.4) HOL=1
1763 IF (MO.EQ.7.AND.DAY.EQ.3.AND.NDAY.EQ.6) HOL=1
1764 IF (MO.EQ.7.AND.CAY.EQ.5.AND.NDAY.EQ.2) HOL=1
1765 IF (MC.EQ.12.AND.CAY.EQ.25) HOL=1
1766 IF (MO.EQ.12.AND.DAY.EQ.24.AND.NDAY.EQ.6) HOL=1
1767 IF (MC.EQ.12.AND.CAY.EQ.26.AND.NDAY.EQ.2) HOL=1
1768 IF (MO.EQ.9.AND.CAY.LT.7.AND.NDAY.EQ.2) HOL=1
1769 IF (MC.EQ.11.AND.CAY.GT.24.AND.NDAY.EQ.5) HOL=1
1770 RETURN
1771 C
1772 END
1773 @FOR,IS A19
1774 SUBRCUTINE MULT (A,B,C,D,AT,BT,CT,DT,N)
1775 C
1776 C *****
1777 C
1778 DIMENSICN A(N),E(N),C(N),D(N)
1779 ATT=A(1)
1780 BTT=B(1)
1781 CTT=C(1)
1782 DTT=D(1)
1783 IF (N.LT.2) GO TO 20
1784 DO 10 J=2,N
1785 AT=ATT*A(J)+ETT*C(J)
1786 BT=ATT*B(J)+BTT*D(J)
1787 CT=CTT*A(J)+CTT*C(J)
1788 DT=CTT*B(J)+DTT*D(J)
1789 ATT=AT
1790 BTT=BT
1791 CTT=CT
1792 DTT=DT
1793 10 GO TO 30
1794 C
1795 20 AT=ATT
1796 BT=BTT
1797 CT=CTT
1798 DT=DTT

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1759 30 RETURN
1800 C
1801 END
1802 @FCR, IS A20
1803 SUBROUTINE OUTSID (X,Y,Z,CR,UX,FO,DB,TIM,QD,QI,QSUN,QSKY,TO,TI,
1804 2 TCNEW,TA,NR)
1805 C
1806 C *****
1807 C
1808 DIMENSION TO(1),TI(1),X(1),Y(1),Z(1)
1809 XNUM=QSUN-QSKY+FO*(DB-TIM)
1810 IF (NR.CT.1) GO TO 40
1811 IF (FO) 10,10,20
1812 10 TCNEW=TO(1)
1813 GO TO 30
1814 C
1815 20 TAM=TA-TIM
1816 TCNEW=(XNUM+UX*TAM)/(UX+FO)
1817 30 CONTINUE
1818 QD=UX*(TAM-TCNEW)
1819 TO(1)=TCNEW
1820 GO TO 80
1821 C
1822 40 SUMZ=0.
1823 SUMY=Y(1)*TI(1)
1824 SUMX=X(1)*TI(1)
1825 SUMXY=0.
1826 DO 50 J=2,NR
1827 SUMY=SUMY+Y(J)*TI(J)
1828 SUMX=SUMX+X(J)*TI(J)
1829 SUMXY=SUMXY+Y(J)*TC(J)
1830 50 SUMZ=SUMZ+Z(J)*TO(J)
1831 XNLW=SLMY-SUMZ+CR*QD+XNUM
1832 TCNEW=XNUM/(Z(1)+FO)
1833 IF (FO) 60,60,70
1834 60 TCNEW=TO(1)
1835 70 TO(1)=TCNEW
1836 SLMZ=SUMZ+Z(1)*TO(1)
1837 SUMXY=SUMXY+Y(1)*TO(1)
1838 QD=SLMY-SUMZ+CR*QC
1839 30 CONTINUE
1840 RETURN
1841 C
1842 END
1843 @FCR, IS A21
1844 SUBROUTINE PSY1 (DB,WB,PB,DP,PV,W,H,V,RH)
1845 C
1846 C *****
1847 C
1848 PVP=PVSF(WB)
1849 IF (CB-WB) 30,30,10
1850 10 WSTAR=0.622*PVP/(PB-PVP)
1851 IF (WB-32.) 20,20,40
1852 20 PV=PVP-5.704E-4*PB*(DB-WB)/1.8
1853 GO TO 50
1854 C
1855 30 PV=PVP
1856 GC TO 50

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1857 C
1858 40 CDB=(DB-32.)/1.8
1859 CWB=(WB-32.)/1.8
1860 HL=597.31+0.4409*CDB-CWB
1861 CH=0.2402+0.4409*WSTAR
1862 EX=(WSTAR-CH*(CDB-CWB)/HL)/0.622
1863 PV=PB*EX/(1.+EX)
1864 50 W=0.622*PV/(PB-PV)
1865 V=0.754*(DB+459.7)*(1+7000*W/4360)/PB
1866 H=0.24*DB+(1061+0.444*CB)*W
1867 IF (PV.LE.0) GO TO 70
1868 IF (CB.NE.WB) GO TO 60
1869 DP=DB
1870 RH=1.
1871 GO TO 70
1872 C
1873 60 CONTINUE
1874 DP=DPF(PV)
1875 RH=PV/PVSF(DB)
1876 70 RETURN
1877 C
1878 END
1879 @FOR, IS A22
1880 SUBROUTINE PSY2 (DB,DP,PB,WB,PV,W,H,V,RH)
1881 C
1882 C *****
1883 C
1884 C THIS SUBROUTINE CALCULATES THE FOLLOWINGS WHEN DRY-BULB TEMPERATURE
1885 C (CB),DEW-PCINT TEMPERATURE(DP),AND BAROMETRIC PRESSURE(PB) ARE GIVEN
1886 C WB WET-BULB TEMPERATURE
1887 C W HUMIDITY RATIO
1888 C H ENTHALPY
1889 C V VOLUME
1890 C PV VAPOR PRESSURE
1891 C RH RELATIVE HUMIDITY
1892 IF (DP-CB) 20,10,10
1893 10 DP=CB
1894 20 PV=PVSF(DP)
1895 PV=PVSF(DP)
1896 PVS=PVSF(DB)
1897 RH=PV/PVS
1898 W=0.622*PV/(PB-PV)
1899 V=0.754*(DB+459.7)*(1+7000*W/4360)/PB
1900 H=0.24*DB+(1061+0.444*CB)*W
1901 IF (H) 30,30,40
1902 30 WB=DP
1903 RETURN
1904 C
1905 40 WB=WB*(H,PB)
1906 RETURN
1907 C
1908 END
1909 @FCR, IS A23
1910 FUNCTION PVSF (X)
1911 C
1912 C *****
1913 C
1914 DIMENSION A(6) /-7.90298,5.02808,-1.3816E-7,11.344,

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1915 2 8.1328E-3,-3.49149/.B(4) /-9.09718,-3.56654,0.876793,0.0060273/
1916 3,P(4)
1917 T=(X+459.688)/1.8
1918 IF (T.LT.273.16) GO TO 10
1919 Z=373.16/T
1920 P(1)=A(1)*(Z-1)
1921 P(2)=A(2)*LOG10(Z)
1922 Z1=A(4)*(1-1/Z)
1923 P(3)=A(3)*(10**Z1-1)
1924 Z1=A(6)*(Z-1)
1925 P(4)=A(5)*(10**Z1-1)
1926 GO TO 20
1927 C
1928 10 Z=273.16/T
1929 P(1)=B(1)*(Z-1)
1930 P(2)=B(2)*LOG10(Z)
1931 P(3)=B(3)*(1-1/Z)
1932 P(4)=LOG10(B(4))
1933 20 SUM=0
1934 DO 30 I=1,4
1935 30 SUM=SUM+P(I)
1936 PVSF=29.921*10**SUM
1937 RETURN
1938 C
1939 END
1940 @FCR,IS A24
1941 SUBROUTINE RESF (XX,YY,ZZ,IRUN)
1942 C
1943 C *****
1944 C
1945 C THIS PROGRAM IS DEVELOPED BY T.KUSUDA OF THE NATIONAL BUREAU OF
1946 C STANDARDS FOR CALCULATING THE THERMAL RESPONSE FACTORS FOR
1947 C COMPOSITE WALLS,FLOORS,ROOFS,BASEMENT WALLS BASEMENT FLOORS
1948 C REAL K(10),G(10),L(10),KG
1949 C DIMENSION X(100),Y(100),Z(100),C(10),D(10),RES(10),RMK(10,6)
1950 C DIMENSION RMKG(6),F(100),XX(100,1),YY(100,1),ZZ(100,1),FF(100,20)
1951 C DELTAT=1.
1952 C IRUN=0
1953 C IN=0
1954 C WRITE (6,200)
1955 10 READ (5,380) NLAYR
1956 C IF (NLAYR.EQ.0) GO TO 190
1957 C IRUN=IRUN+1
1958 C IF (NLAYR.GT.10) GO TO 190
1959 C IF (NLAYR.EQ.0) GO TO 30
1960 C DO 20 I=1,NLAYR
1961 20 READ (5,380) L(I),K(I),D(I),C(I),RES(I)
1962 C WRITE (6,210)
1963 C IF (IN.EQ.2) GO TO 40
1964 C READ K,RHO, AND C OF GROUND IF IN=1
1965 C FOLLOWINGS ARE GROUND THERMAL CONDUCTIVITY, DENSITY AND SP.HT IF
1966 C IN=2, OTHERWISE THE SAME PROPERTIES OF THE INTERNAL SLAB
1967 30 IF (IN.NE.0) READ (5,380) KG,DG,CG
1968 C AG THERMAL DIFFUSIVITY OF EARTH
1969 C IF (IN.NE.0) AG=KG/CG/DG
1970 C IF (NLAYR.EQ.0) GO TO 90
1971 C IF (IN.EC.2) READ (5,320) (RMKG(J),J=1,6)
1972 40 DO 50 I=1,NLAYR

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1973 50 READ (5,320) (RMK(I,J),J=1,6)
1974 C IF (IN.EQ.1) READ (5,320) (RMKG(J),J=1,6)
1975 C DO 80 I=1,NLAYR
1976 C IF (L(I)) 70,60,70
1977 60 G(I)=0.
1978 C K(I)=1./RES(I)
1979 C GO TO 80
1980 70 G(I)=K(I)/C(I)/D(I)
1981 80 CONTINUE
1982 90 CONTINUE
1983 C CALL RESPTK (K,L,C,AG,KG,X,Y,Z,NLAYR,DELTAT,NRT,CR,UT,IN,F)
1984 C WRITE (6,230) IRUN
1985 C WRITE (6,340)
1986 C WRITE (6,240)
1987 C WRITE (6,250)
1988 C WRITE (6,220)
1989 C IF (NLAYR.EQ.0) GO TO 120
1990 C IF (IN.EQ.2) WRITE (6,350) KG,DG,CG,(RMKG(J),J=1,4)
1991 C DO 110 I=1,NLAYR
1992 C IF (L(I)) 110,100,110
1993 100 K(I)=0.
1994 110 WRITE (6,260) I,L(I),K(I),D(I),C(I),RES(I),(RMK(I,J),J=1,6)
1995 C IF (IN.EC.1) WRITE (6,350) KG,DG,CG,(RMKG(J),J=1,6)
1996 120 WRITE (6,280) DELTAT
1997 C WRITE (6,270) UT
1998 C WRITE (6,290)
1999 C WRITE (6,220)
2000 C IF (IN.NE.0) GO TO 140
2001 C WRITE (6,300)
2002 C XX(1,IRUN)=FLOAT(NRT)
2003 C YY(1,IRUN)=FLOAT(NRT)
2004 C ZZ(1,IRUN)=FLOAT(NRT)
2005 C XX(2,IRUN)=CR
2006 C YY(2,IRUN)=CR
2007 C ZZ(2,IRUN)=CR
2008 C XX(NRT+3,IRUN)=UT
2009 C DO 130 N=1,NRT
2010 C XX(N+2,IRUN)=X(N)
2011 C YY(N+2,IRUN)=Y(N)
2012 C ZZ(N+2,IRUN)=Z(N)
2013 C JN=N-1
2014 130 WRITE (6,310) JN,X(N),Y(N),Z(N)
2015 C GC TC 180
2016 C
2017 140 WRITE (6,360)
2018 C IF (IN.EQ.1) GO TO 160
2019 C IF (IN.EQ.2) GO TO 160
2020 C XX(1,IRUN)=FLOAT(NRT)
2021 C XX(2,IRUN)=CR
2022 C XX(NRT+3,IRUN)=UT
2023 C DO 150 N=1,NRT
2024 C JN=N-1
2025 C X(N)=-X(N)
2026 C XX(N+2,IRUN)=X(N)
2027 150 WRITE (6,370) JN,X(N)
2028 C GC TC 180
2029 C
2030 160 DO 170 N=1,NRT

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2031      JN=N-1
2032      FF(N+2,IRUN)=F(N)
2033 170    WRITE (6,370) JN,F(N)
2034      FF(1,IRUN)=FLOAT(NRT)
2035      FF(2,IRUN)=CR
2036      FF(NRT+3,IRUN)=UT
2037 180    WRITE (6,220)
2038      WRITE (6,220)
2039      WRITE (6,330) CR
2040      GG TO 10
2041      C
2042 190    RETURN
2043      C
2044      C
2045 200    FORMAT (28H DATA SHEET NO 6 N/L,K,P,C,R)
2046 210    FORMAT (43H DATA SHEET NO 7: DESCRIPTION OF EACH LAYER)
2047 220    FORMAT (2H0 )
2048 230    FORMAT (7H1 IRF=I10)
2049 240    FORMAT (77H0 LAYER      L(I)      K(I)      (I)      C(I)      RES(
2050 2I)  DESCRIPTION      )
2051 250    FORMAT (70H NO
2052 2      OF LAYERS)
2053 260    FORMAT (1I6,1F11.3,1F10.3,1F10.2,1F10.3,1F8.2,2X,6A4)
2054 270    FORMAT (58H0          THERMAL CONDUCTANCE
2055 2      U=1F7.3)
2056 280    FORMAT (49H0          TIME INCREMENT DT=1F3.0)
2057 290    FORMAT (50H0          RESPONSE FACTORS)
2058 300    FORMAT (/70H          J          X          Y
2059 2      Z)
2060 310    FORMAT (1I17,1F23.4,2F15.4)
2061 320    FORMAT (6A4)
2062 330    FORMAT (44H0          COMMON RATIO CR=1F7.5)
2063 340    FORMAT (50H0 WALL COMPOSITION      )
2064 350    FORMAT (1F27.3,1F10.2,1F10.3,10X,6A4)
2065 360    FORMAT (50H0          J          F          )
2066 370    FORMAT (1I24,1F21.5)
2067 380    FORMAT ( )
2068      C
2069      END
2070 @FOR, IS A25
2071      SUBROUTINE RESFX (X,Y,Z,XX,YY,ZZ,NR,CR,UT,NEXP)
2072      C
2073      C *****
2074      C
2075      DIMENSION XX(100,10),YY(100,10),ZZ(100,10)
2076      DIMENSION X(10,100),Y(10,100),Z(10,100),NR(10),CR(10),UT(10)
2077      TEST=1.E-6
2078      DO 10 K=1,10
2079      DO 10 J=1,100
2080      XX(J,K)=0
2081      YY(J,K)=0
2082 10      ZZ(J,K)=0
2083      CALL RESF (XX,YY,ZZ,IRUN)
2084      DO 50 K=1,NEXP
2085      I=K
2086      IF(YY(5,K))11,11,12
2087 11      YY(3,K)=0.
2088      YY(4,K)=0.

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2089      YY(5,K)=0.
2090 12      CONTINUE
2091      IF (K.GT.IRUN) GO TO 50
2092      X(I,1)=XX(3,K)
2093      Y(I,1)=YY(3,K)
2094      Z(I,1)=ZZ(3,K)
2095      NR(I)=XX(1,K)
2096      CR(I)=XX(2,K)
2097      JJJ=NR(I)+3
2098      UT(I)=XX(JJJ,K)
2099      NMAX=NR(I)
2100      J1=1
2101      DO 40 J=2,NMAX
2102      J3=J+2
2103      J2=J+1
2104      X(I,J)=XX(J3,K)-XX(J2,K)*CR(I)
2105      Y(I,J)=YY(J3,K)-YY(J2,K)*CR(I)
2106      IF (ABS(X(I,J))-TEST) 20,20,30
2107 20      NR(I)=J
2108      GO TO 50
2109 30      CONTINUE
2110      Z(I,J)=ZZ(J3,K)-ZZ(J2,K)*CR(I)
2111      JK=J
2112 40      CONTINUE
2113 50      CONTINUE
2114      RETURN
2115      C
2116      C
2117      END
2118 @FCR, IS A26
2119      SUBROUTINE RESPTK (K,L,G,AG,KG,X,Y,Z,NL,DT,NR,CR,U,IS,F)
2120      C
2121      C *****
2122      C
2123      DIMENSION K(10),L(10),G(10),X(100),Y(100),Z(100),AP(10),BP(10),
2124 2 CP(10),DP(10),A(10),B(10),C(10),D(10),ZR1(3),ZR2(3),RB(3),RAP(3),
2125 3 ROOT(100),RA(3,100),ZRK(3,100),RX(100),RY(100),AZ(100),F(100)
2126      REAL K,L,KG
2127      PI=4.*ATAN(1.)
2128      ZN=1.
2129 5      M3=3
2130      IF (IS.NE.1) GO TO 10
2131      ZL=KG/10.
2132      UY=100./AG/DT
2133      CALL GPF (UY,ZL,AZ)
2134      IF (IS.EQ.1.AND.NL.EQ.0) GO TO 360
2135 10      CALL ABCD2 (0.,K,L,G,AX,BX,CX,DX,NL)
2136      RB(1)=DX
2137      RB(2)=1.
2138      RB(3)=AX
2139      U=1./BX
2140      DO 20 I=1,NL
2141      PX=0
2142      CALL ABCDP2 (PX,K(I),L(I),G(I),AP(I),BP(I),CP(I),DP(I))
2143 20      CALL ABCD2 (PX,K(I),L(I),G(I),A(I),B(I),C(I),D(I),1)
2144      IF (NL.LT.2) GO TO 30
2145      CALL DERVT (A,B,C,D,AP,BP,CP,DP,APP,BPP,PPP,DPP,NL)
2146      GO TO 40

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2147 25 ZN=ZN+1.
2148 IF(ZN.GT.4.)GO TO 510
2149 GC TC 5
2150 C
2151 30 APP=AP(1)
2152 BPP=EP(1)
2153 CFF=CP(1)
2154 DPP=CP(1)
2155 4C RAP(1)=DPP
2156 RAP(2)=0.
2157 RAF(3)=AFP
2158 DO 50 I=1,3
2159 C1=RAP(1)/BX/DT
2160 C2=RE(I)*BFP/BX/BX/DT
2161 ZR2(I)=-C1+C2
2162 50 ZR1(I)=-ZR2(I)+RE(I)/BX
2163 C ROOTS OF B(P)=0.
2164 NMAX=40
2165 TESTMX=40.
2166 PX=0.001
2167 DPC=0.001/DT
2168 DLX=C.0001
2169 N=0
2170 60 DL=DPO
2171 CALL ABCD2 (PX,K,L,G,AX,BX,CX,DX,NL)
2172 70 PXP=PX+DL
2173 CALL ABCD2 (PXP,K,L,G,AXP,BXP,CXP,DXP,NL)
2174 IF (BX*BXP) 90,110,80
2175 8C PX=PXP
2176 BX=BXP
2177 TESTX=PX*DT
2178 IF (TESTX-TESTMX) 70,170,170
2179 90 IF (DL-DLX) 140,14C,100
2180 100 DL=DL/2.
2181 GO TO 70
2182 C
2183 110 IF (BX) 130,120,130
2184 120 RXX=PX
2185 GO TO 150
2186 C
2187 130 RXX=PXP
2188 GO TO 150
2189 C
2190 140 AE=AE(BX/BXP)
2191 RXX=(PX+AB*PXP)/(1.+AB)
2192 150 N=N+1
2193 RCCT(N)=RXX
2194 IF (N.GT.1)DPC=(ROOT(N)-ROOT(N-1))/ZN
2195 NRT=N
2196 PX=RXX+DLX
2197 TESTX=RXX*DT
2198 IF (TESTX-TESTMX) 160,160,170
2199 160 IF (N.LT.NMAX) GO TO 60
2200 170 CONTINUE
2201 IF (ROOT(NRT)-100.) 190,180,180
2202 180 NRT=NRT-1
2203 190 DO 250 JJ=1,NRT
2204 PX=ROOT(JJ)

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2205 DO 200 J=1,NL
2206 CALL ABCD2 (PX,K(J),L(J),G(J),A(J),B(J),C(J),D(J),1)
2207 200 CALL ABCD2 (PX,K(J),L(J),G(J),AP(J),BP(J),CP(J),DP(J))
2208 CALL ABCD2 (PX,K,L,G,AX,BX,CX,DX,NL)
2209 IF (NL.LT.2) GO TO 210
2210 CALL DERTV (A,B,C,D,AP,BP,CP,DP,APP,BPP,CPP,DPP,NL)
2211 GO TO 220
2212 210 APP=AP(1)
2213 BPP=EP(1)
2214 CFF=CP(1)
2215 DPP=CP(1)
2216 220 PY=BFF*FX*FX*DT
2217 RA(1,JJ)=DX/PY
2218 RA(2,JJ)=1./PY
2219 RA(3,JJ)=AX/PY
2220 PZ=PX*DT
2221 IF (PZ-20.) 240,240,230
2222 230 RX(JJ)=0.
2223 RY(JJ)=2E.E16
2224 GO TO 250
2225 240 RX(JJ)=EXP(-PZ)
2226 RY(JJ)=(1.-EXP(PZ))*2
2227 250 CONTINUE
2228 DO 260 JJ=1,NRT
2229 DO 260 M=1,M3
2230 ZR1(M)=RA(M,JJ)*RX(JJ)+ZR1(M)
2231 260 ZR2(M)=RA(M,JJ)*(RX(JJ)*RX(JJ)-2.*RX(JJ))+ZR2(M)
2232 II=1
2233 III=2
2234 IF (ZR1(2).LT.0) ZR1(2)=0.
2235 DO 270 M=1,M3
2236 ZRK(M,1)=ZR1(M)
2237 270 ZRK(M,2)=ZR2(M)
2238 NT=100
2239 DO 310 N=3,NT
2240 NR=N
2241 DO 280 M=1,M3
2242 280 ZRK(M,N)=0.
2243 DO 290 M=1,M3
2244 DO 290 JJ=1,NRT
2245 IF (RX(JJ).LE.0.)GO TO 288
2246 APCWER=ALOG10(RX(JJ))
2247 NDERFL=ABS(APOWER*N)
2248 IF (NDERFL.LE.37)GO TO 285
2249 288 PZ=0.
2250 GO TO 290
2251 285 PZ=(RX(JJ))*N
2252 290 ZRK(M,N)=ZRK(M,N)+PZ*RY(JJ)*RA(M,JJ)
2253 IF (ZRK(1,N-1)*ZRK(1,N-2)) 300,320,300
2254 300 IF (N.LT.5) GO TO 310
2255 TEST1=ZRK(1,N)/ZRK(1,N-1)
2256 TEST2=ZRK(1,N-1)/ZRK(1,N-2)
2257 TEST3=AE(TEST1-TEST2)
2258 IF (TEST3-0.001) 320,320,310
2259 310 CONTINUE
2260 320 DO 330 N=1,NR
2261 X(N)=ZRK(1,N)
2262 Y(N)=ZRK(2,N)

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2262 IF(Y(N).LT.-1.0E-04)GO TO 25
2264 330 Z(N)=ZRK(3,N)
2265 CR=TEST2
2266 IF (X(3)) 350,340,350
2267 340 CR=0.
2268 350 CONTINUE
2269 IF (IS.EQ.2) GO TO 480
2270 IF (IS.NE.1) GO TO 500
2271 360 IF (NL.EQ.0) GO TO 420
2272 GF=2*KG/SQRT(DT*AG*PI)
2273 IF (NR.LT.50) GO TO 380
2274 DO 370 J=50,NR
2275 ZJ=J
2276 370 AZ(J)=GF*(SQRT(ZJ)-2.*SQRT(ZJ-1.)+SQRT(ZJ-2.))
2277 NRR=NR
2278 GO TO 400
2279 C
2280 380 DO 390 J=NR,50
2281 Z(J+1)=Z(J)*CR
2282 X(J+1)=X(J)*CR
2283 390 Y(J+1)=Y(J)*CR
2284 NRR=50
2285 400 DO 410 J=1,NRR
2286 410 F(J)=X(J)-Y(J)*Y(J)/(Z(J)+AZ(J))
2287 NR=NRR
2288 GO TO 440
2289 C
2290 420 DO 430 J=1,NR
2291 430 F(J)=AZ(J)
2292 440 CONTINUE
2293 CR1=1.
2294 DO 460 J=1,50
2295 CR=F(J+1)/F(J)
2296 TESTCR=ABS(CR-CR1)
2297 IF (TESTCR-0.001) 470,470,450
2298 450 CR1=CR
2299 JJ=J-1
2300 460 CONTINUE
2301 470 NR=J
2302 CR=CR1
2303 GO TO 500
2304 C
2305 480 CONTINUE
2306 DO 490 J=1,NR
2307 F(J)=X(J)+Z(J)-2.*Y(J)
2308 JJ=J-1
2309 490 CONTINUE
2310 500 RETURN
2311 510 WRITE(6,520)
2312 520 FORMAT(27H ERROR IN SUBROUTINE RESPTK)
2313 C
2314 C
2315 END
2316 @FOR, IS A27
2317 SUBROUTINE RMRT (HEATG,HLCG,HEATX,HLCX,HEATIS,HLCIS,IW,IL,FC,ISC)
2318 C
2319 C *****
2320 C

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2321 C FC: CCFRECTION FACTORFOR THE HEAT LOST TO THE SURROUNDINGS
2322 C ISC:SHADING COEFFICIENT INDEX IF ISC=0 EXTERNAL SHADING
2323 C OTHERWISE INTERNAL SHADING
2324 DIMENSION HEATG(2),HLCG(2),HEATX(2),HLCX(2),HEATIS(2),HLCIS(2)
2325 DIMENSION AGO(3),AG1(3),AXO(3),AIS1(4,3),AIS2(4,3),B1(3),AX1(3)
2326 DATA AGO /0.187,0.197,0.224/, AG1 /-0.097,-0.067,-0.044/
2327 DATA B1 /-0.91,-0.87,-0.82/
2328 DATA AXO /0.676,0.681,0.703/, AX1 /-0.566,-0.551,-0.523/
2329 DATA (AIS1(1,J),J=1,3) /0.53,0.53,0.53/
2330 DATA (AIS2(1,J),J=1,3) /-0.44,-0.40,-0.35/
2331 DATA (AIS1(2,J),J=1,3) /0.59,0.59,0.59/
2332 DATA (AIS2(2,J),J=1,3) /-0.50,-0.46,-0.41/
2333 DATA (AIS1(3,J),J=1,3) /0.87,0.87,0.87/
2334 DATA (AIS2(3,J),J=1,3) /-0.78,-0.74,-0.69/
2335 DATA (AIS1(4,J),J=1,3) /0.50,0.50,0.50/
2336 DATA (AIS2(4,J),J=1,3) /-0.41,-0.37,-0.32/
2337 HLCG(2)=FC*(AGO(IW)*HEATG(2)+AG1(IW)*HEATG(1))-B1(IW)*HLCG(1)
2338 HLCX(2)=FC*(AXO(IW)*HEATX(2)+AX1(IW)*HEATX(1))-B1(IW)*HLCX(1)
2339 HLCIS(2)=FC*(AIS1(IL,IW)*HEATIS(2)+AIS2(IL,IW)*HEATIS(1))-B1(IW)*H
2340 2LCIS(1)
2341 IF (ISC.EQ.0) RETURN
2342 HLCG(2)=FC*(AGO(IW)*HEATG(2)+AX1(IW)*HEATG(1))-B1(IW)*HLCG(1)
2343 RETURN
2344 C
2345 END
2346 @FCR, IS A28
2347 SLBRoutine RMTMK (V,TIF,QL,TA,NEXP,NX,ITK)
2348 C
2349 C *****
2350 C
2351 CCMCN /CC/ X(10,100),Y(10,100),Z(10,100),ITYPE(30),IMT(30),
2352 2 IRF(30),ABSP(30),U(30),H(30),HI(30),A(30),UT(30),TOS(30,48),
2353 3 TIS(30,48),G(30,30),TOY(48),DB(24),QLITX(24,3),QEQUX(24,3),
2354 4 QGCLP(24,3),QDCPS(24),QLITE(24),QEQUP(24),QI(30),CR(30),NR(30),
2355 5 QGLAS(30,24),ITHST,UENDW,AZW(30),SHADE(30),RMDBS(24),RMDBW(24),
2356 6 SHD(30),UCELNG
2357 DIMENSION AA(30,30),BB(30),TT(30),TIF(30),A2(30,30),B2(30),B3(30),
2358 2 GSUM(30),V(15)
2359 TS=V(1)
2360 CFML=V(2)
2361 CFMS=V(3)
2362 RRDOM=V(4)
2363 RCELG=V(5)
2364 RRCCML=V(12)
2365 TIM=V(6)
2366 QCMAX=V(7)
2367 QHMAX=V(8)
2368 TUL=V(9)
2369 TTL=V(10)
2370 TSET=V(11)
2371 HR=V(13)
2372 MET=V(14)
2373 DBNX=DB(NX)-TIM
2374 TU=TS-TIM
2375 NEXP2=NEXP+1
2376 DO 10 I=1,NEXP
2377 BB(I)=0.
2378 B2(I)=0.

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2379      DO 10 J=1,NEXP
2380      A2(I,J)=0.
2381 10    AA(I,J)=0.
2382      SHG=0.
2383      HSUM=0.
2384      ASUM=0.
2385      ASUMT=0.
2386      DO 70 I=1,NEXP
2387      NRR=NR(I)
2388      SHG=SHG+QGLAS(I,NX)*A(I)
2389      ASUMT=ASUMT+A(I)
2390      GSUM(I)=0.
2391      DO 20 J=1,NEXP
2392 20    GSUM(I)=GSUM(I)+G(I,J)
2393      IF (ITYPE(I).NE.3) ASUM=ASUM+A(I)
2394      IF (MET.NE.0) GSUM(I)=HR
2395      HSUM=HSUM+H(I)*A(I)
2396      IR=IRF(I)
2397      CRX=CR(I)
2398      IF(UENDW.NE.0..AND.ITYPE(I).EQ.1)NRR=1
2399      XTEMP=X(IR,1)
2400      YTEMP=Y(IR,1)
2401      ZTEMP=Z(IR,1)
2402      IF (NRR.GE.2) GO TO 30
2403      X(IR,1)=UT(I)
2404      Y(IR,1)=UT(I)
2405      CRX=0.
2406      Z(IR,1)=UT(I)
2407 30    AA(I,I)=X(IR,1)+HI(I)+GSUM(I)
2408      DO 40 J=1,NEXP
2409      IF (I.EQ.J) GC TO 40
2410      AA(I,J)=-G(I,J)
2411 40    CONTINUE
2412      AA(I,NEXP2)=-HI(I)
2413      SUMY=Y(IR,1)*TOS(I,1)
2414      SUMX=0.
2415      IF (NRR.LT.2) GC TO 60
2416      DO 50 J=2,NRR
2417      SUMY=SUMY+Y(IR,J)*TOS(I,J)
2418 50    SUMX=SUMX+X(IR,J)*TIS(I,J)
2419 60    CONTINUE
2420      B3(I)=SUMY-CFX*QI(I)-SUMX
2421      AA(NEXP2,I)=A(I)*HI(I)
2422      X(IR,1)=XTEMP
2423      Y(IR,1)=YTEMP
2424 70    Z(IR,1)=ZTEMP
2425      SHX=SHG/ASUM
2426      QLTEMP=QLITE(NX)*(1-RCELG)*RROOML+(QOCPS(NX)+QEQU(NX))*RRROOM
2427      QLX=QLTEMP/ASUMT
2428      DO 90 I=1,NEXP
2429      SHF=SHX
2430      QLT=QLX
2431      IF (ITYPE(I).EQ.3.OR.ITYPE(I).EQ.7) SHF=0.
2432      IF (ITYPE(I).EQ.7) QLT=0.
2433 90    BB(I)=B3(I)+SHF+QLT
2434      AA(NEXP2,NEXP2)=-1.08*(CFML+CFMS)-HSUM
2435      JK=1
2436      NEXP3=NEXP2+1

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2437      DC 100 I=1,NEXP2
2438      DO 100 J=1,NEXP2
2439 100   A2(I,J)=AA(I,J)
2440      SUM1=(QOCPS(NX)+QEQU(NX))*(1.-RRROOM)
2441      SUM2=QLITE(NX)*(1.-RRROOML)*(1.-RCELG)
2442      SUM3=0.
2443      SUM=1.08*(CFML*DBNX+CFMS*TU)+SUM1+SUM2
2444      BB(NEXP2)=-SUM
2445      IF (ITHST.NE.0.AND.ITK.EQ.0) GO TO 150
2446 110   BB(NEXP2)=-SLM-SUM3
2447      IF (MET.EQ.0) GO TO 130
2448      SUM4=0.
2449      SUM5=0.
2450      DO 120 I=1,NEXP
2451      SUM4=SUM4+(HI(I)*A(I)*BB(I))/AA(I,I)
2452      SUM5=SUM5+H(I)*A(I)*(HI(I)+HR)/AA(I,I)
2453 120   CONTINUE
2454      TT(NEXP2)=(SUM4-BB(NEXP2))/(-AA(NEXP2,NEXP2)-SUM5)
2455      DO 121 I=1,NEXP
2456 121   TT(I)=(HI(I)+HR)*TT(NEXP2)+BB(I)/AA(I,I)
2457      GO TO 140
2458      C
2459 130   CONTINUE
2460      CALL SOLVP (NEXP2,NEXP3,AA,BB,TT,30)
2461 140   TA=TT(NEXP2)+TIM
2462      IF (ITHST.EQ.0.AND.ITK.EQ.1) GO TO 190
2463      IF (JK.EQ.2) GO TO 190
2464      GO TO 200
2465      C
2466 150   DG 160 I=1,NEXP2
2467 160   B2(I)=BB(I)-AA(I,NEXP2)*(TA-TIM)
2468      IF (MET.EQ.0) GO TO 180
2469      DO 170 I=1,NEXP
2470      TT(I)=(HI(I)+HR)*(TA-TIM)+BB(I)/AA(I,I)
2471 170   CONTINUE
2472      GO TO 190
2473      C
2474 180   CONTINUE
2475      CALL SOLVP (NEXP,NEXP2,A2,B2,TT,30)
2476 190   CONTINUE
2477      QL=SUM-1.08*(CFML+CFMS)*(TA-TIM)
2478      GO TO 240
2479      C
2480 200   IF (TA-TUL) 210,220,220
2481 210   IF (TA-TLL) 230,230,190
2482 220   TA=TUL
2483      GO TO 150
2484      C
2485 230   TA=TLL
2486      GO TO 150
2487      C
2488 240   SUMG=0.
2489      DO 250 I=1,NEXP
2490      K=IRF(I)
2491      XK1=X(K,1)
2492      TIS(I,1)=TT(I)
2493      TEST=ABS(TT(I))
2494      IF (TEST.GT.100.) GO TO 310

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2495      IF(K.EQ.10)X(K,1)=UT(I)
2496      QI(I)=X(K,1)*TT(I)-B3(I)
2497      IF (ITYPE(I).EQ.7) QI(I)=0.
2498      IF (UENDW.NE.0..AND.ITYPE(I).EQ.1) QI(I)=UT(I)*(TT(I)-TOS(1,1))
2499      TIF(I)=TT(I)+TIM
2500      IF (ITYPE(I).EQ.7) TIF(I)=TA
2501      SUMQ=SUMQ+A(I)*HI(I)*(TA-TIF(I))
2502      X(K,1)=XK1
2503 250  CONTINUE
2504      QL=-QL+SUMQ
2505      IF (ITHST.NE.0.OR.ITK.NE.0) GO TO 300
2506      IF (JK.EQ.2) GG TC 300
2507      IF (QL) 260,300,280
2508 260  QLTEST=ABS(QL)
2509      IF (QLTEST-QCMAX) 300,300,270
2510 270  SUM3=-QCMAX
2511      JK=2
2512      GO TO 110
2513  C
2514 280  IF (QL-QHMAX) 300,300,290
2515 290  SUM3=QHMAX
2516      JK=2
2517      GO TO 110
2518  C
2519 300  RETURN
2520  C
2521 310  CONTINUE
2522      WRITE (6,330)
2523      DD 320 I=1,NEXP2
2524 320  WRITE (6,340) (A2(I,J),J=1,NEXP2),B2(I),TT(I)
2525      RETURN
2526  C
2527  C
2528 330  FORMAT (109H ERROR IN THE RMTMK ROUTINE: MATRIX ELEMENTS ARE LISTE
2529      2D IN THE FOLLOWING ORDER: AA(I,J),J=1,NEXP2,B(I),T(I))
2530 340  FORMAT (12F10.3)
2531  C
2532      END
2533 2FOR, IS A29
2534      SUBROUTINE RCCMX (NEXP,NS,NW,NN,NE,H)
2535  C
2536  C *****
2537  C
2538      DIMENSION NEXP(4)
2539      COMMON /CC/ X(10,100),Y(10,100),Z(10,100),ITYPE(30),IHT(30),
2540      2 IRF(30),ABSP(30),U(30),HT(30),HI(30),A(30),V(30),TOS(30,48),
2541      3 TIS(30,48),G(30,30),TDY(48),DB(24),QLITX(24,3),QEOUX(24,3),
2542      4 QOCLP(24,3),QOCPS(24),QLITE(24),QEQUP(24),QI(30),CR(30),NR(30),
2543      5 QGLAS(30,24),ITHST,UENDW,AZW(30),SHADE(30),RMDBS(24),RMDBW(24),
2544      6 SHD(30),UCELNG
2545      REAL L,FS(6,6)
2546      CCKMGN /SHDW/ SHACW(30,15)
2547  C
2548  C *****
2549  C * * *
2550  C * * *
2551  C * *****
2552  C * * SOUTH FACING *

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2553  C (W)* * WALL *(H)
2554  C * * *
2555  C *****
2556  C (L)
2557  C
2558  C
2559  C NS = NUMBER OF HEAT TRANSFER SURFACES IN THE SOUTH WALL
2560  C NW = NUMBER OF HEAT TRANSFER SURFACES IN THE WEST WALL
2561  C NN = NUMBER OF HEAT TRANSFER SURFACES IN THE NORTH WALL
2562  C NE = NUMBER OF HEAT TRANSFER SURFACES IN THE EAST WALL
2563  C NEXP = TOTAL NUMBER OF HEAT TRANSFER SURFACES = 2+NS+NW+NN+NE
2564      WRITE (6,50)
2565      READ (5,110) NNEXP
2566      NS=NNEXP(1)
2567      NW=NNEXP(2)
2568      NN=NNEXP(3)
2569      NE=NNEXP(4)
2570      NEXP=2+NS+NW+NN+NE
2571  C L = ROOM LENGTH ALONG THE SOUTH WALL
2572  C W = ROOM WIDTH ALONG THE WEST WALL
2573  C H = ROOM CEILING HEIGHT
2574      READ (5,110) L,W,H
2575      CALL FCTR (L,W,H,FS)
2576      NS=NS+1
2577      NW=NS+NW
2578      NN=NW+NN
2579      NE=NE+NN
2580      AS=L*H
2581      AW=W*H
2582      AN=AS
2583      AE=AW
2584      AR=L*W
2585      AF=AR
2586      WRITE (6,60)
2587      DC 10 I=1,NEXP
2588  C READ FOLLOWING DATA IN THE ORDER OF CEILING, SOUTH WALL, WEST
2589  C WALL, NORTH WALL, EAST WALL, FLOOR,
2590      READ (5,110) ITYPE(I),IRF(I),A(I),AZW(I),U(I),SHADE(I),ABSP(I),SHD
2591      2(I)
2592  C READ SHADOW INFORMATION
2593      READ (5,110) (SHADW(I,J),J=1,7)
2594      READ (5,110) (SHADW(I,J),J=8,15)
2595 10  CONTINUE
2596      DO 20 I=1,NEXP
2597      IF (I.EQ.1) M=1
2598      IF (I.GT.1.AND.I.LE.NS) M=2
2599      IF (I.GT.NS.AND.I.LE.NW) M=3
2600      IF (I.GT.NW.AND.I.LE.NN) M=4
2601      IF (I.GT.NN.AND.I.LE.NE) M=5
2602      IF (I.EQ.NEXP) M=6
2603      DO 20 J=1,NEXP
2604      IF (J.EQ.1) G(I,J)=FS(M,1)
2605      IF (J.GT.1.AND.J.LE.NS) G(I,J)=FS(M,2)*A(J)/AS
2606      IF (J.GT.NS.AND.J.LE.NW) G(I,J)=FS(M,3)*A(J)/AW
2607      IF (J.GT.NW.AND.J.LE.NN) G(I,J)=FS(M,4)*A(J)/AN
2608      IF (J.GT.NN.AND.J.LE.NE) G(I,J)=FS(M,5)*A(J)/AE
2609      IF (J.EQ.NEXP) G(I,J)=FS(M,6)
2610 20  CONTINUE

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2611 RETURN
2612 C
2613 C * * * * *
2614 C
2615 ENTRY RMOUT
2616 C
2617 C * * * * *
2618 C
2619 WRITE (6,70) NEXP,NNEXP,L,W,H
2620 WRITE (6,80)
2621 DO 30 I=1,6
2622 WRITE (6,100) (FS(I,J),J=1,6)
2623 30 CONTINUE
2624 WRITE (6,90)
2625 DO 40 I=1,NEXP
2626 WRITE (6,100) (G(I,J),J=1,NEXP)
2627 40 CONTINUE
2628 RETURN
2629 C
2630 C
2631 50 FORMAT (36H DATA SHEET NO 10: NS,NW,NN,NE,L,W,H)
2632 60 FORMAT (68H DATA SHEET 11 AND 12: ROOM SURFACE DATA AND EXTERIOR S
2633 2URFACE SHADW)
2634 70 FORMAT (///5X24HTHIS ROOM HAS A TOTAL OFI3,23H HEAT TRANSFER SURFA
2635 2CES//15X15HSOUTH WALL HAS I3,9H SURFACES//15X15HWEST WALL HAS I3,
2636 39H SURFACES//15X15+NORTH WALL HAS I3,9H SURFACES//15X15HEAST WALL
2637 4 HAS I3,9H SURFACES//5X30HROOM LENGTH ALONG SOUTH WALL =F7.2//5X30
2638 SHRCOM WIDTH ALONG WEST WALL =F7.2//5X30HROOM CEILING HEIGHT
2639 6 =F7.2//)
2640 80 FORMAT (24H1 BASIC SHAPE FACTORS//)
2641 90 FORMAT (////31H RADIATION EXCHANGE FACTORS//)
2642 100 FORMAT (20F6.3)
2643 110 FORMAT ( )
2644 C
2645 END
2646 @FCR, IS A30
2647 SUBROUTINE SHADOW (SHDX,PHI,COSZ,SHRAT)
2648 C
2649 C *****
2650 C
2651 DIMENSION SHDX(20)
2652 FL=SHDX(1)
2653 HT=SHDX(2)
2654 FP=SHDX(3)
2655 AW=SHDX(4)
2656 BWL=SHDX(5)
2657 BWR=SHDX(6)
2658 D=SHDX(7)
2659 FP1=SHDX(8)
2660 A1=SHDX(9)
2661 B1=SHDX(10)
2662 C1=SHDX(11)
2663 FP2=SHDX(12)
2664 A2=SHDX(13)
2665 B2=SHDX(14)
2666 C2=SHDX(15)
2667 WAZI=SHDX(16)
2668 C THIS PROGRAM CALCULATES SHADOW CAST BY OVERHANG AND SIDE FINS

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2669 C THIS PROGRAM HAS BEEN DEVELOPED BY TSENG-YAO SUN
2670 C PHI....SOLAR AZIMUTH ANGLE
2671 C COSZ...CCSINE OF SOLAR ZENITH ANGLE
2672 C SHRAT..SHADE RATIO:RATIO OF THE SUNLIT AREA TO THE TOTAL WINDOW AR
2673 C HT.....WINDOW HEIGHT
2674 C FL.....WINDOW WIDTH
2675 C FP.....DEPTH OF THE OVERHUNG
2676 C AW.....DISTANCE FROM TOP OF THE WINDOW TO THE OVERHUNG
2677 C BWL....DISTANCE OF THE OVERHUNG EXTENDED BEYOND THE LEFT EDGE OF T
2678 C BWR....DISTANCE OF THE OVERHUNG EXTENDED BEYOND THE RIGHT EDGE OF
2679 C D.....DEPTH OF VERTICAL PROJECTION AT THE END OF THE OVERHUNG
2680 C FP1....DEPTH OF THE LEFT FIN
2681 C A1.....DISTANCE OF THE LEFT FIN EXTENDED ABOVE THE TOP OF THE WIND
2682 C B1.....DISTANCE FROM THE LEFT EDGE OF THE WINDOW TO THE LEFT FIN
2683 C C1.....DISTANCE OF THE LEFT FIN STOP SHORT ABOVE THE BOTTOM OF THE
2684 C FP2....DEPTH OF THE RIGHT FIN
2685 C A2.....DISTANCE OF THE RIGHT FIN EXTENDED ABOVE THE TOP OF THE WIN
2686 C B2.....DISTANCE FROM THE RIGHT EDGE OF THE WINDOW TO THE RIGHT FIN
2687 C C2.....DISTANCE OF THE RIGHT FIN STOP SHORT ABOVE THE BOTTOM OF TH
2688 C WAZI...WINDOW AZIMUTH ANGLE
2689 SHRAT=1.
2690 A=AW
2691 H=HT
2692 GAMMA=PHI-WAZI
2693 COSG=COS(GAMMA)
2694 IF (COSG) 10,10,20
2695 10 SHRAT=0.
2696 GO TO 910
2697 C
2698 20 CONTINUE
2699 SBETA=CCSZ
2700 IF (SBETA) 10,10,30
2701 30 SING=SIN(GAMMA)
2702 VERT=SBETA/SQRT(1.-SBETA*SBETA)/COSG
2703 HCRIZ=ABS(SING)/COSG
2704 TCETA=VERT/HCRIZ
2705 IF (GAMMA) 50,40,40
2706 C -----SUN ON LEFT
2707 40 B=BWL
2708 GO TO 60
2709 C -----SUN ON RIGHT
2710 50 B=BWR
2711 60 ARSHF=0.
2712 AREAV=0.
2713 ARSIF=0.
2714 AREAC=0.
2715 AREA1=0.
2716 ARSH1=0.
2717 FL3=0.
2718 H3=0.
2719 H1=H
2720 FL1=FL
2721 K=1
2722 L=1
2723 T1=FP*VERT
2724 FM1=FP*HCRIZ
2725 IF (FP) 430,430,70
2726 70 T=T1

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2727      FM=FM1
2728      AB=B*TCETA
2729      UG=(FL+B)*TCETA
2730      DE=(H+A)/TCETA
2731 C      -----HCRIZCN TAL OVERTUNG AREA
2732      IF (T-A) 330,330,80
2733 80     IF (AB-A) 200,200,90
2734 90     IF (DE-B) 180,180,100
2735 100    IF (FM-B) 170,170,110
2736 110    IF (DE-(FL+B)) 140,140,120
2737 120    IF (FM-(FL+B)) 150,130,130
2738 C      -----HORIZ 9
2739 130    AREAD=FL*(0.5*(AB+UG)-A)
2740      GC TC 430
2741 C
2742 140    IF (T-(H+A)) 150,160,160
2743 C      -----HCRIZ 7
2744 150    AREAC=(T-A)*FL-((FM-B)**2)*TCETA*0.5
2745 C      L=2
2746      GO TO 270
2747 C      -----HORIZ 8
2748 160    AREAG=H*FL-(DE-B)**2*TCETA*0.5
2749      GO TO 430
2750 C      -----HORIZ 3
2751 170    AREAD=FL*(T-A)
2752 C      L=2
2753      GO TO 300
2754 C
2755 180    IF (T-(H+A)) 170,190,190
2756 C      -----HORIZ 2
2757 190    AREAD=H*FL
2758      GO TO 900
2759 C
2760 200    IF (UG-A) 330,330,210
2761 210    IF (DE-(FL+B)) 240,240,220
2762 220    IF (FM-(FL+B)) 260,230,230
2763 C      -----HORIZ 6
2764 230    AREAD=(UG-A)**2/TCETA*0.5
2765 C      GO TO 430
2766 C
2767 240    IF (T-(H+A)) 260,250,250
2768 C      -----HORIZ 5
2769 250    AREAD=H*(FL-(A+0.5*H)/TCETA+B)
2770      GC TC 430
2771 C      -----HORIZ 4
2772 260    AREAC=(T-A)*(FL+B-FM*(1.+A/T)*0.5)
2773 C      L=2
2774 C      -----VERT PROJ AREA
2775 270    FL3=FL+B-FM
2776      IF (T+D-(H+A)) 280,280,290
2777 C      -----VERT 8
2778 280    H3=D
2779      GO TO 420
2780 C      -----VERT 9
2781 290    H3=H+A-T
2782      GO TO 420
2783 C
2784 300    FL3=FL

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2785      IF (T+D-(H+A)) 320,320,310
2786 C      -----VERT 7
2787 310    H3=H+A-T
2788      AREAV=H3*FL3
2789      GO TO 900
2790 C      -----VERT 6
2791 320    H3=D
2792      GO TO 420
2793 C
2794 330    IF (T+D-A) 430,430,340
2795 340    IF (FM-B) 390,390,350
2796 350    IF (FM-(FL+B)) 360,430,430
2797 360    FL3=FL+B-FM
2798      IF (T+D-(H+A)) 380,380,370
2799 C      -----VERT 5
2800 370    H3=H
2801      GO TO 420
2802 C      -----VERT 4
2803 380    H3=T+D-A
2804      GO TC 420
2805 C
2806 390    IF (T+D-(H+A)) 410,400,400
2807 C      -----VERT 2
2808 400    AREAV=H*FL
2809      GO TO 900
2810 C      VERT 3
2811 410    H3=T+D-A
2812      FL3=FL
2813 420    AREAV=FL3*H3
2814 C      -----SIDE FIN AND SHORT SIDE FIN
2815 C      -----SIDE FIN AREA1 ARSIF
2816 430    IF (GAMMA) 450,900,440
2817 440    FPF=FP1
2818      AF=A1
2819      BF=B1
2820      CX=C1
2821      GO TO 460
2822 C
2823 450    FFF=FP2
2824      AF=A2
2825      BF=B2
2826      CX=C2
2827 460    IF (FPF) 900,900,470
2828 470    T=FPF*VERT
2829      FM=FPF*HCRIZ
2830      AF1=AF
2831      IF (AREAD) 530,530,480
2832 C      -----TEST FOR CVERLAP OF FIN AND OVERHUNG SHADOW
2833 480    AT=A+(BF-B)*TCETA
2834      IF (AT-AF) 490,530,530
2835 C      -----OVERLAP EXISTS..L=2 IF OVERHUNG SHADOW HAS HORIZ EDGE IN WIN
2836 490    GO TO (510,500), L
2837 C      -----TEST FOR TYPE OF OVERLAP
2838 500    IF ((FM-BF)-(FM1-B)) 510,520,520
2839 C      -----SET L=1,SHADOW INTERSECT ON INCLINED EDGE OF OVERHUNG SHADOW
2840 C      -----FIN SHADOW IS BELOW INCLINED EDGE OF OVERHUNG SHADOW
2841 510    AF=AT
2842      L=1

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2843      GO TO 530
2844 C      -----L IS 2, HORIZ EDGE OF OVERHUNG SHADOW-PORTION ABOVE HORIZ E
2845 C      -----NOT IN OVERHUNG SHADOW IS FIN SHADOW
2846 520    AREA1=FL*(T1-A)-AREA0
2847 C      -----RESET TO CALC FIN SHADOW BELOW HORIZ EDGE OF OVHNG SHADOW
2848      AF=T1-A+AF1
2849      H=H+AF1-AF
2850 C      -----SHADOW OF FIN (K=1 ON GLASS K=2 ON VERT PROJ SHADOW)
2851 530    AB=BF*TCETA
2852      UG=(FL+BF)*TCETA
2853      DE=(H+AF)/TCETA
2854      DJ=CX/TCETA
2855      IF (FM-BF) 800,800,540
2856 540    IF (AB-AF) 550,660,660
2857 550    IF (UG-AF) 630,630,560
2858 560    IF (T-AF) 640,640,570
2859 570    IF (UG-(H+AF)) 600,600,580
2860 580    IF (T-(H+AF)) 620,590,590
2861 C      -----FIN 9
2862 590    AREA1=H*((AF+H*0.5)/TCETA-BF)+AREA1
2863      GC TO 730
2864 C
2865 600    IF (FM-(FL+BF)) 620,610,610
2866 C      -----FIN 8
2867 610    AREA1=H*FL-(UG-AF)**2/TCETA*0.5+AREA1
2868      GO TO 730
2869 C      -----FIN 7
2870 620    AREA1=(FM-BF)*H-(T-AF)**2/TCETA*0.5+AREA1
2871      GO TO 770
2872 C
2873 630    IF (FM-(FL+BF)) 640,640,650
2874 C      -----FIN 3
2875 640    AREA1=H*(FM-BF)+AREA1
2876      GO TO 770
2877 C      -----FIN 2
2878 650    AREA1=H*FL+AREA1
2879      GO TO 730
2880 C
2881 660    IF (DE-BF) 800,800,670
2882 670    IF (UG-(H+AF)) 710,710,680
2883 680    IF (T-(H+AF)) 700,690,690
2884 C      -----FIN 6
2885 690    AREA1=(DE-BF)**2*TCETA*0.5+AREA1
2886      GC TO 730
2887 C      -----FIN 4
2888 700    AREA1=(FM-BF)*(H+AF-(T+AB)*0.5)+AREA1
2889      GO TO 770
2890 C
2891 710    IF (FM-(FL+BF)) 700,720,720
2892 C      -----FIN 5
2893 720    AREA1=FL*(H-(BF+FL*0.5)*TCETA+AF)+AREA1
2894 C      -----SHORT SIDE FIN ARSH1, ARSHF
2895 730    IF (DJ-BF) 800,800,740
2896 740    IF (DJ-(FL+BF)) 760,760,750
2897 C      -----SHORT 3
2898 750    ARSH1=-FM*(CX-(BF+FL/2.)*TCETA)
2899      GO TO 800
2900 C      -----SHORT 4

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2901 760    ARSH1=-CX-AB)**2/TCETA*0.5
2902      GO TO 800
2903 C
2904 770    IF (DJ-BF) 800,800,780
2905 780    IF (DJ-FM) 760,760,790
2906 C      -----SHORT 2
2907 790    ARSH1=-FM*(CX-(T+AB)*0.5)
2908 800    GO TO (E20,810), K
2909 C
2910 810    ARSH1=-ARSH1
2911      AREA1=-AREA1
2912 820    ARSHF=ARSHF+ARSH1
2913      ARSIF=ARSIF+AREA1
2914      GC TO (E30,900), K
2915 C
2916 830    IF (AREA1) 900,900,840
2917 C      -----RESET PARAMETERS TO DEDUCT FIN SHADOW OVERLAP ON VERT PROJ
2918 840    K=2
2919      AREA1=0.
2920      ARSH1=0.
2921      BBF=BF
2922      BF=FM1-B+BF
2923      IF (BF) 850,860,860
2924 850    BF=BBF
2925 860    IF (HT+A-T1-D) 850,890,870
2926 870    CX=CX-(HT+A-T1-D)
2927      IF (CX) 880,890,890
2928 880    CX=0.
2929 890    AF=T1-A+AF
2930      H=H3
2931      FL=FL3
2932      GO TO 530
2933 C      ----- SHADED AREA ARSHA
2934 900    ARSHA=AREA0+AREA1+ARSHF+ARSIF
2935      SHRAT=(FL1*H1-ARSHA)/(FL1*H1)
2936      FL=FL1
2937 910    CONTINUE
2938      RETURN
2939 C
2940      END
2941 2FCR, IS A31
2942      SLBROUTINE SHG (SH)
2943 C
2944 C      *****
2945 C
2946      DIMENSION SH(20)
2947 C      SH(1)=INTENSITY OF DIRECT NORMAL SOLAR RADIATION
2948 C      SH(2)=INTENSITY OF DIFFUSE SKY RADIATION
2949 C      SH(3)=INTENSITY OF GROUND REFLECTED DIFFUSE RADIATION
2950 C      SH(4)=COSINE OF INCIDENCE OF DIRECT SOLAR RADIATION
2951 C      SH(5)=FORM FACTOR BETWEEN THE WINDOW AND THE SKY
2952 C      SH(6)=FORM FACTOR BETWEEN THE WINDOW AND THE GROUND
2953 C      SH(7)=THERMAL RESISTANCE AT OUTSIDE SURFACE
2954 C      SH(8)=THERMAL RESISTANCE AT THE AIR SPACE (DOUBLE GLAZING)
2955 C      SH(9)=THERMAL RESISTANCE AT THE INNER SURFACE
2956 C      SH(10)=SUNLIT AREA FACTOR
2957 C      SH(11)=SHADING COEFFICIENT ,NON-ZERO VALUE WILL BE GIVEN ONLY
2958 C      WHEN THE WINDOW IS SHADED BY DRAPES OR BLINDS OR IF IT HAS

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2959 C      AN INTERPANE SEPARATION OF MORE THAN 1-INCH
2960 C      SH(12)=TRANSMISSION FACTOR FOR DIRECT RADIATION
2961 C      SH(13)=TRANSMISSION FACTOR FOR DIFFUSE RADIATION
2962 C      SH(14)=ABSORPTION FACTOR FOR DIRECT RADIATION (OUTER PANE)
2963 C      SH(15)=ABSORPTION FACTOR FOR DIRECT RADIATION (INNER PANE)
2964 C      SH(16)=ABSORPTION FACTOR FOR DIFFUSE RADIATION (OUTER PANE)
2965 C      SH(17)=ABSORPTION FACTOR FOR DIFFUSE RADIATION (INNER PANE)
2966 C      SH(18)=SCALAR HEAT GAIN
2967 C      COMMON /SOL/ LAT,LCNG,TZN,WAZ,WT,CN,DST,LPIR,S(35)
2968 C      REAL LAT,LCNG,NI,NO
2969 C      NI=(SH(7)+SH(8))/(SH(7)+SH(8)+SH(9))
2970 C      NO=(SH(7))/(SH(7)+SH(8)+SH(9))
2971 C      D=S+(10)*SH(1)*SH(4)*(SH(12)+NO*SH(14)+NI*SH(15))
2972 C      DD=(SH(2)*SH(5)+SH(3)*SH(6))*(SH(13)+NO*SH(16)+NI*SH(17))
2973 C      IF (SH(11)) 20,10,20
2974 10     SH(18)=D+DD
2975 C      GO TO 30
2976 C
2977 20     SH(18)=(D+DD)*SH(11)
2978 30     RETURN
2979 C
2980 C      END
2981 @FOR,IS A32
2982     SUBROUTINE SOLVP (M,N,C,D,X,I)
2983 C
2984 C      *****
2985 C
2986 C      THIS IS A ROUTINE FOR SOLVING SIMULTANEOUS LINEAR EQUATIONS
2987 C      THE ROUTINE WAS DEVELOPED BY B.A. PEAVY OF NBS
2988 C      ROUTINE FAILS WHEN ANY OF THE DIAGONAL ELEMENTS IS ZERO
2989 C      DIMENSION A(100,101),C(1,1),D(1),X(1)
2990 C      DO 10 IX=1,M
2991 C      DO 10 IY=1,M
2992 10     A(IX,IY)=C(IX,IY)
2993 C      DO 20 IZ=1,M
2994 20     A(IZ,N)=D(IZ)
2995 C      L=1
2996 30     AA=A(L,L)
2997 C      DO 40 K=L,N
2998 40     A(L,K)=A(L,K)/AA
2999 C      DO 60 K=1,M
3000 C      IF (K.EQ.L) GO TO 60
3001 C      AA=-A(K,L)
3002 C      DO 50 IA=L,N
3003 50     A(K,IA)=A(K,IA)+AA*A(L,IA)
3004 60     CONTINUE
3005 C      L=L+1
3006 C      IF (L.LE.M) GO TO 30
3007 C      DO 70 IP=1,M
3008 70     X(IP)=A(IP,N)
3009 C      RETURN
3010 C
3011 C      END
3012 @FOR,IS A33
3013     SUBROUTINE SUN
3014 C
3015 C      *****
3016 C

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3017     DIMENSION A0(5) /,302,-,0002,368.44,,1717.0,0905/ ,
3018 2 A1(5) /-22.53,,4197,24.52,-,0344,-,0410/ ,
3019 3 A2(5) /-,229,-,3.2265,-1.14,,0032,,0073/ ,
3020 4 A3(5) /-,243,-,0903,-1.09,,0024,,0015/ ,
3021 5 B1(5) /3.851,-7.351,,58,-,0043,-,0034/ ,B2(5) /,002,-9.3912,-,18,
3022 6 0,,0.0004/ ,B3(5) /-,055,-,3361,,28,-,0008,-,0006/
3023     REAL LAT,LATD,LCNG,MERID,LCND
3024     COMMON /SOL/ LAT,LCNG,TZN,WAZ,WY,CN,DST,LPIR,S(35)
3025 C      S(1)= LATITUDE,DEGREES(+NORTH,-SOUTH)
3026 C      S(2)= LONGITUDE,DEGREES(+WEST,-EAST)
3027 C      S(3)= TIME ZONE NUMBER
3028 C      STANDARD TIME          DAYLIGHT SAVING TIME
3029 C      ATLANTIC                4                3
3030 C      EASTERN                 5                4
3031 C      CENTRAL                 6                5
3032 C      MOUNTAIN                7                6
3033 C      PACIFIC                 8                7
3034 C      S(4)= DAYS(FROM START OF YEAR)
3035 C      S(5)= TIME, HOUR AFTER MIDNIGHT)
3036 C      S(6)= DAYLIGHT SAVING TIME INDICATOR
3037 C      S(7)= GROUND REFLECTIVITY
3038 C      S(8)= CLEARNESS NUMBER
3039 C      S(9)= WALL AZIMUTH ANGLE, DEGREES FROM SOUTH
3040 C      S(10)=WALL TILT ANGLE, DEGREES FROM HORIZON
3041 C      S(11)=SUN RISE TIME (HOURS AFTER MIDNIGHT)
3042 C      S(12)=SUN SET TIME
3043 C      S(13)=COSZ DIRECTION COSINES
3044 C      S(14)=CCSN DIRECTION COSINES
3045 C      S(15)=COS(S) DIRECTION COSINES)
3046 C      S(16)=ALPHA DIRECTION COSINES NORMAL TO SURFACE
3047 C      S(17)=BETA
3048 C      S(18)=GAMMA
3049 C      S(19)=COS(ETA)COSINE OF INCIDENCE ANGLE
3050 C      S(20)=SOLAR ALTITUDE ANGLE
3051 C      S(21)=SCALAR AZIMUTH ANGLE
3052 C      S(22)=DIFFUSE SKY RADIATION ON HORIZONTAL SURFACE
3053 C      S(23)=DIFFUSE GROUND REFLECTED RADIATION
3054 C      S(24)=DIRECT NORMAL RADIATION
3055 C      S(25)=TOTAL SOLAR RADIATION INTENSITY
3056 C      S(26)=DIFFUSE SKY RADIATION INTENSITY
3057 C      S(27)=GROUND REFLECTED DIFFUSE RADIATION INTENSITY
3058 C      S(28)=SUN DECLINATION ANGLE,DEGREES
3059 C      S(29)=EQUATION OF TIME ,HOURS
3060 C      S(30)=A SOLAR FACTOR
3061 C      S(31)= SOLAR FACTOR
3062 C      S(32)= SOLAR FACTOR
3063 C      S(33)= CLOUD COVER MODIFIER
3064 C      S(34) INTENSITY OF DIRECT SOLAR RADIATION ON SURFACE
3065 C      S(35) HOUR ANGLE,DEGREE
3066     PI=3.1415927
3067     X=2*PI/366.*S(4)
3068     C1=COS(X)
3069     C2=COS(2*X)
3070     C3=CCS(3*X)
3071     S1=SIN(X)
3072     S2=SIN(2*X)
3073     S3=SIN(3*X)
3074     DO 10 K=1,5

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3075 KS=(K-1)+28
3076 10 S(KS)=A0(K)+A1(K)*C1+A2(K)*C2+A3(K)*C3+B1(K)*S1+B2(K)*S2+B3(K)*S3
3077 S(29)=S(29)/60.
3078 LATD=S(1)
3079 LONG=S(2)
3080 MERID=15*S(3)
3081 LOND=LONG-MERID
3082 Y=S(28)*PI/180.
3083 YY=LATD*PI/180.
3084 HP=-TAN(Y)*TAN(YY)
3085 TR=12/PI*ACOS(HP)
3086 S(11)=(12-TR)-S(29)+LOND/15.
3087 S(12)=24.-S(11)
3088 H=15*(S(5)-12+S(2)+S(29)-S(6))-S(2)
3089 S(35)=H
3090 S13=SIN(YY)*SIN(Y)+COS(YY)*COS(Y)*COS(H*PI/180.)
3091 S(13)=S13
3092 HP1=180.*ACOS(HP)/PI
3093 X1=ABS(HP1)
3094 X2=ABS(H)
3095 IF (X1-X2) 130,20,20
3096 20 S(14)=COS(Y)*SIN(H*PI/180.)
3097 S(15)=SQRT(1.-S(13)*S(13)-S(14)*S(14))
3098 STEST=S(15)
3099 STEST1=COS(H*PI/180.)-TAN(Y)/TAN(YY)
3100 IF (STEST1) 40,30,30
3101 30 S(15)=STEST
3102 GO TO 50
3103 C
3104 4C S(15)=-STEST
3105 50 S(20)=ASIN(S(13))
3106 IF (S(15)) 70,60,60
3107 6C S(21)=ASIN(S(14)/COS(S(20)))
3108 GO TO 80
3109 C
3110 70 S(21)=PI-ASIN(S(14)/COS(S(20)))
3111 80 S(20)=180.*S(20)/PI
3112 S(21)=180.*S(21)/PI
3113 S(24)=S(30)*S(8)*S(33)*EXP(-S(31)/S(13))
3114 S(22)=S(32)*S(24)/S(8)/S(8)
3115 S(23)=S(7)*(S(22)+S(24)*S(13))
3116 WY=S(10)*PI/180.
3117 S(16)=COS(WY)
3118 WA=S(9)*PI/180.
3119 S(16)=COS(WY)
3120 S(17)=SIN(WA)*SIN(WY)
3121 S(18)=COS(WA)*SIN(WY)
3122 S(19)=S(16)*S(13)+S(17)*S(14)+S(18)*S(15)
3123 S(34)=S(24)*S(19)
3124 Y=0.45
3125 IF (S(19)+0.2) 100,100,90
3126 90 Y=0.85+0.437*S(19)+0.313*S(19)**2
3127 100 IF (S(19)) 110,110,120
3128 110 S(19)=0.
3129 S(34)=0.
3130 120 CONTINUE
3131 S(26)=S(22)*Y
3132 S(27)=S(23)*(1-S(16))/2.

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3133 S(25)=S(34)+S(26)+S(27)
3134 GO TO 150
3135 C
3136 130 DO 140 J=14,26
3137 140 S(J)=0.
3138 S(34)=0
3139 150 RETURN
3140 C
3141 END
3142 @FCR, IS A34
3143 SUBROUTINE TAR (TR)
3144 C
3145 C *****
3146 C
3147 REAL A1(6) /0.01154,0.77674,-3.94657,8.57881,-8.38135,3.01188/
3148 REAL A2(6) /0.01636,1.40783,-6.79030,14.37378,-13.83357,4.92439/
3149 REAL A3(6) /0.01837,1.92497,-8.89134,18.40197,-17.48648,6.17544/
3150 REAL A4(6) /0.01902,2.35417,-10.4715,21.24322,-19.95978,6.99964/
3151 REAL A5(6) /0.01712,3.50839,-13.8639,26.34330,-23.84846,8.17372/
3152 REAL A6(6) /0.01406,4.15958,-15.0628,27.18492,-23.88518,8.03650/
3153 REAL A7(6) /0.01153,4.55946,-15.4329,26.70568,-22.87993,7.57795/
3154 REAL A8(6) /0.00962,4.81911,-15.4714,25.86516,-21.69106,7.08714/
3155 REAL T1(6) /-0.00885,2.71235,-0.62062,-7.07329,9.75995,-3.89922/
3156 REAL T2(6) /-0.01114,2.39371,0.42978,-8.98262,11.51798,-4.52054/
3157 REAL T3(6) /-0.01200,2.13036,1.13833,-10.07925,12.44161,-4.83285/
3158 REAL T4(6) /-0.01218,1.90950,1.61391,-10.64872,12.83698,-4.95199/
3159 REAL T5(6) /-0.01056,1.29711,2.28615,-10.37132,11.95884,-4.54880/
3160 REAL T6(6) /-0.00835,0.92766,2.15721,-8.71429,9.87152,-3.73328/
3161 REAL T7(6) /-0.00646,0.68256,1.82499,-6.95325,7.80647,-2.94454/
3162 REAL T8(6) /-0.00496,0.51043,1.47607,-5.41985,6.00546,-2.28162/
3163 REAL A01(6) /0.01407,1.06226,-5.59131,12.15034,-11.78092,4.20070/
3164 REAL A02(6) /0.01819,1.86277,-9.24831,19.49443,-18.56094,6.53940/
3165 REAL A03(6) /0.01905,2.47900,-11.7427,24.14037,-22.64299,7.89954/
3166 REAL A04(6) /0.01862,2.96400,-13.4870,27.13020,-25.11877,8.68895/
3167 REAL A05(6) /0.01423,4.14384,-16.66709,31.30484,-27.81955,9.36959/
3168 REAL A06(6) /0.01056,4.71447,-17.33454,30.91781,-26.63898,8.79495/
3169 REAL A07(6) /0.00819,5.01768,-17.21228,29.46388,-24.76915,8.05040/
3170 REAL A08(6) /0.00670,5.18781,-16.84820,27.90292,-22.99619,7.38140/
3171 REAL A11(6) /0.00228,0.34559,-1.19908,2.22336,-2.05287,0.72376/
3172 REAL A12(6) /0.00123,0.29788,-0.92256,1.58171,-1.40040,0.48316/
3173 REAL A13(6) /0.00061,0.26017,-0.72713,1.14950,-0.97138,0.32705/
3174 REAL A14(6) /0.00035,0.22974,-0.58381,0.84626,-0.67666,0.22102/
3175 REAL A15(6) /-0.00009,0.15049,-0.27590,0.25618,-0.12919,0.02859/
3176 REAL A16(6) /-0.00016,0.10579,-0.15035,0.06487,0.02759,-0.02317/
3177 REAL A17(6) /-0.00015,0.07717,-0.09059,0.00050,0.06711,-0.03394/
3178 REAL A18(6) /-0.00012,0.05746,-0.05878,-0.01855,0.06837,-0.03191/
3179 REAL TD1(6) /-0.00401,0.74050,7.20350,-20.11763,19.68824,-6.74585/
3180 REAL TD2(6) /-0.00438,0.57818,7.42065,-20.26848,19.79706,-6.79619/
3181 REAL TD3(6) /-0.00428,0.45797,7.41367,-19.92004,19.40969,-6.66603/
3182 REAL TD4(6) /-0.00401,0.36698,7.27324,-19.29364,18.75408,-6.43968/
3183 REAL TD5(6) /-0.00279,0.16468,6.17715,-15.84811,15.28302,-5.23666/
3184 REAL TD6(6) /-0.00192,0.08180,4.94753,-12.43481,11.92495,-4.07787/
3185 REAL TD7(6) /-0.00136,0.04419,3.87529,-9.59069,9.16022,-3.12776/
3186 REAL TD8(6) /-0.00098,0.02576,3.00400,-4.33834,6.98747,-2.38328/
3187 DIMENSION TR(9),A(8,6),T(8,6),AO(8,6),AI(8,6),TD(8,6)
3188 C TR(1)= TRANSMISSION FACTOR ,DIRECT
3189 C TR(2)= TRANSMISSION FACTOR ,DIFFUSE
3190 C TR(3)= ABSORPTION FACTOR ,DIRECT, OUTER

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3191 C TR(4)= .DIFFUSE,OUTER
3192 C TR(5)= .DIRECT,INNER
3193 C TR(6)= .DIFFUSE,INNER
3194 C TR(7)= COSINE OF INCIDENT ANGLE
3195 C TR(8)=TYPE CF GLASS
3196 C TR(9)=ID CODE FOR THE GLAZING
3197 C ID =1 SINGLE GLAZING
3198 C ID =2 DOUBLE GLAZING
3199 DO 10 J=1,6
3200 A(1,J)=A1(J)
3201 A(2,J)=A2(J)
3202 A(3,J)=A3(J)
3203 A(4,J)=A4(J)
3204 A(5,J)=A5(J)
3205 A(6,J)=A6(J)
3206 A(7,J)=A7(J)
3207 A(8,J)=A8(J)
3208 T(1,J)=T1(J)
3209 T(2,J)=T2(J)
3210 T(3,J)=T3(J)
3211 T(4,J)=T4(J)
3212 T(5,J)=T5(J)
3213 T(6,J)=T6(J)
3214 T(7,J)=T7(J)
3215 T(8,J)=T8(J)
3216 AO(1,J)=AO1(J)
3217 AO(2,J)=AO2(J)
3218 AO(3,J)=AC3(J)
3219 AO(4,J)=AO4(J)
3220 AO(5,J)=AC5(J)
3221 AO(6,J)=AO6(J)
3222 AO(7,J)=AO7(J)
3223 AO(8,J)=AO8(J)
3224 AI(1,J)=AI1(J)
3225 AI(2,J)=AI2(J)
3226 AI(3,J)=AI3(J)
3227 AI(4,J)=AI4(J)
3228 AI(5,J)=AI5(J)
3229 AI(6,J)=AI6(J)
3230 AI(7,J)=AI7(J)
3231 AI(8,J)=AI8(J)
3232 TD(1,J)=TD1(J)
3233 TD(2,J)=TD2(J)
3234 TD(3,J)=TD3(J)
3235 TD(4,J)=TD4(J)
3236 TD(5,J)=TD5(J)
3237 TD(6,J)=TD6(J)
3238 TD(7,J)=TD7(J)
3239 10 TD(8,J)=TD8(J)
3240 ETA=TR(7)
3241 L=TR(8)
3242 ID=TR(9)
3243 IF (ID.EQ.2) GO TO 30
3244 TR(1)=T(L,1)
3245 TR(2)=T(L,1)/2.
3246 TR(3)=A(L,1)
3247 TR(4)=A(L,1)/2.
3248 DO 20 J=2,6

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3249 TR(1)=TR(1)+T(L,J)*(ETA**(J-1))
3250 TR(2)=TR(2)+T(L,J)/(J+1)
3251 TR(3)=TR(3)+A(L,J)*(ETA**(J-1))
3252 20 TR(4)=TR(4)+A(L,J)/(J+1)
3253 TR(5)=0
3254 TR(6)=0
3255 GO TO 50
3256 C
3257 30 TR(1)=TD(L,1)
3258 TR(2)=TD(L,1)/2.
3259 TR(3)=AC(L,1)
3260 TR(4)=AO(L,1)/2.
3261 TR(5)=AI(L,1)
3262 TR(6)=AI(L,1)/2.
3263 DC 40 J=2,6
3264 X=ETA**(J-1)
3265 TR(1)=TR(1)+TD(L,J)*X
3266 TR(2)=TR(2)+TD(L,J)/(J+1)
3267 TR(3)=TR(3)+AO(L,J)*X
3268 TR(4)=TR(4)+AO(L,J)/(J+1)
3269 TR(5)=TR(5)+AI(L,J)*X
3270 40 TR(6)=TR(6)+AI(L,J)/(J+1)
3271 50 TR(2)=2*TR(2)
3272 TR(4)=2*TR(4)
3273 TR(6)=2*TR(6)
3274 RETURN
3275 C
3276 END
3277 @FOR, IS A35
3278 SUBROUTINE TEMPSH (MONTH, JJ, NK, RMDBS, RMDBW, RMDBWO, RMDBSO, TA)
3279 C
3280 C *****
3281 C
3282 DIMENSION RMDBS(24), RMDBW(24)
3283 IF (MONTH.GE.6.AND.MONTH.LE.9) GO TO 20
3284 IF (JJ.GT.1) GO TO 10
3285 TA=RMDBW(NK)
3286 GO TO 40
3287 C
3288 10 TA=RMDBWC
3289 GO TO 40
3290 C
3291 20 IF (JJ.GT.1) GO TO 30
3292 TA=RMDBS(NK)
3293 GC TC 40
3294 C
3295 30 TA=RMDBSO
3296 40 CONTINUE
3297 RETURN
3298 C
3299 END
3300 @FOR, IS A36
3301 FUNCTION WBF (H, PB)
3302 C
3303 C *****
3304 C
3305 C THIS PROGRAM APPROXIMATES THE WET-BULB TEMPERATURE WHEN
3306 C ENTHALPY IS GIVEN

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3307 IF (H) 30,30,10
3308 10 Y=LOG(H)
3309 IF (H.GT.11.758) GC TO 20
3310 WBF=0.6041+3.4841*Y+1.3601*Y*Y+0.97307*Y*Y*Y
3311 GC TC 100
3312 C
3313 20 WBF=30.9185-39.68200*Y+20.5841*Y*Y-1.758*Y*Y*Y
3314 GO TO 100
3315 C
3316 30 WB1=150.
3317 PV1=PVSF(WB1)
3318 W1=0.622*PV1/(PB-PV1)
3319 X1=0.24*WB1+(1061+0.444*WB1)*W1
3320 Y1=H-X1
3321 40 WB2=WB1-1
3322 PV2=PVSF(WB2)
3323 W2=0.622*PV2/(PB-PV2)
3324 X2=0.24*WB2+(1061+0.444*WB2)*W2
3325 Y2=H-X2
3326 IF (Y1*Y2) 50,60,50
3327 50 WB1=WB2
3328 Y1=Y2
3329 GO TO 40
3330 C
3331 60 IF (Y1) 80,70,80
3332 70 WBF=WB1
3333 GO TO 100
3334 C
3335 80 WBF=WB2
3336 GC TC 100
3337 C
3338 90 Z=ABS(Y1/Y2)
3339 WBF=(WB2*Z+WB1)/(1+Z)
3340 100 RETURN
3341 C
3342 END
3343 @FCR,IS A37
3344 COMPILER (FLC=ABS)
3345 C
3346 C
3347 C *****
3348 C
3349 SUBPOLINE WD (MM)
3350 C
3351 C *****
3352 C
3353 INTEGER INCM(176,2),TAPE1,TAPE2
3354 COMMON TAPE1,TAPE2,NC1,NO2,INPUT(1100)
3355 MT=TAPE2
3356 IF (MM.EQ.0) MT=TAPE1
3357 NL=176
3358 IF (MM.EQ.0) NL=83
3359 NM=NO1
3360 NO1=NO2
3361 NO2=NM
3362 IF (NC1.NE.NO2) GC TO 10
3363 NO2=2
3364 CALL NTRAN (MT,24,26)

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3365 CALL NTRAN (MT,2,NL,INCM(1,1),L,22)
3366 10 IF (L.LT.0) GO TO 30
3367 CALL NTRAN (MT,2,NL,INCM(1,NO2),L,22)
3368 DO 20 J=1,NL
3369 DO 20 K=1,6
3370 JJ=(K-1)*6
3371 JK=(J-1)*6+K
3372 20 INPUT(JK)=FLC(JJ,6,INCM(J,NC1))
3373 GO TO 40
3374 C
3375 30 WRITE (6,50) L
3376 STOP
3377 C
3378 40 CONTINUE
3379 RETURN
3380 C
3381 C
3382 50 FORMAT (20H0 INPUT ERROR L=1110)
3383 C
3384 END
3385 @FCR,IS A38
3386 SUBROUTINE WDX (IW)
3387 C *****
3388 C
3389 C DIMENSION IDATA(10) /054,06,07,010,011,012,013,014,015,016/ ,
3390 2 JDATA(10) /C60,C61,062,C63,064,C65,066,067,070,071/ ,
3391 3 KDATA(10) /030,C17,020,021,022,023,024,025,026,027/
3392 DATA KX /041/
3393 DO 10 KK=1,10
3394 IF (IW.EQ.IDATA(KK)) GC TO 20
3395 IF (IW.EQ.JDATA(KK)) GO TO 20
3396 IF (IW.EQ.KDATA(KK)) GC TO 30
3397 CONTINUE
3398 10 IF (IW.EQ.KX) GO TO 40
3399 IW=1CCCC000
3400 GO TO 50
3401 C
3402 C
3403 20 IW=KK-1
3404 GO TO 50
3405 C
3406 30 IW=-(KK-1)
3407 GO TO 50
3408 C
3409 40 IW=10
3410 50 RETURN
3411 C
3412 END
3413 @FCR,IS A39
3414 SUBROUTINE WINTER (A,U,ITYPE,NEXP,CFMWT,DBIN,DBWT,UG,TGW,RHI,RHO,
3415 *UENDW,UCELNG,AENDW,ATCHT,AIRCHG)
3416 C *****
3417 C
3418 C DIMENSION A(30),U(30),ITYPE(30)
3419 CALL CBRW (DBWT,RHO,W0)
3420 CALL DBRW (DBIN,RHI,WI)
3421 DT=DBIN-CBWT
3422

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3423 DW=WI-WO
3424 QWINTS=1.08*CFMWT*DT
3425 QWINTL=4.5*CFMWT*DW*1060.
3426 DO 20 I=1,NEXP
3427 IF (ITYPE(I).EQ.6.OR.ITYPE(I).EQ.7) GO TO 20
3428 IF (UCELNG.EQ.0..OR.ITYPE(I).NE.1)GO TO 5
3429 X=U(I)*A(I)+1.08*AIRCHG*ATCMT/60.+UENDW*AENDW
3430 Y=UCELNG*A(I)
3431 TATTIC=(X*DBWT+Y*CBIN)/(X+Y)
3432 QWINTS=QWINTS+Y*(DBIN-TATTIC)
3433 GO TO 20
3434 5 IF (ITYPE(I).NE.5) GO TO 10
3435 QWINTS=QWINTS+UG*A(I)*(DBIN-TGW)
3436 GO TO 20
3437 1C CONTINUE
3438 QWINTS=QWINTS+U(I)*A(I)*DT
3439 20 CONTINUE
3440 TOTAL=QWINTS+QWINTL
3441 WRITE (6,30) QWINTS,QWINTL,TOTAL
3442 RETURN
3443 C
3444 C
3445 30 FORMAT (//30H HEATING LOAD IN BTU PER HOUR /26H SENSIBLE
3446 2LCAD = F10.0/26H LATENT LOAD = F10.0/36H -----
3447 3-----/26H TOTAL LOAD = F10.0//)
3448 C
3449 END
3450 2FOR, IS A40
3451 FUNCTION WKDAY (YR,MO,CAY)
3452 C
3453 C *****
3454 C
3455 C WKDAY=1 SUNDAY
3456 C WKDAY=2 MONDAY
3457 C WKDAY=3 TUESDAY
3458 C WKDAY=4 WEDNESDAY
3459 C WKDAY=5 THURSDAY
3460 C WKDAY=6 FRIDAY
3461 C WKDAY=7 SATURDAY
3462 INTEGER YR,DAY,WKDAY,TCAY,FSTDAY
3463 DIMENSION FSTDAY(12) /31,59,90,120,151,181,212,243,273,304,334,
3464 2 365/
3465 N=YR/4
3466 ND=N-485
3467 IY=2
3468 IF (ND.EQ.0) GO TO 40
3469 IF (NC.LT.0) GO TO 10
3470 IADD=2
3471 GO TO 20
3472 C
3473 10 ND=-ND
3474 IADD=-2
3475 20 DO 30 J=1,ND
3476 IY=IY-IADD
3477 IF (IY.GT.7) IY=IY-7
3478 IF (IY.EQ.0) IY=7
3479 IF (IY.LT.0) IY=IY+7
3480 30 CONTINUE

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3481 40 MD=YR-N*4
3482 IF (MD.EQ.0) IWK=IY
3483 IF (MC.EQ.1) IWK=IY+2
3484 IF (MC.EQ.2) IWK=IY+3
3485 IF (MC.EQ.3) IWK=IY+4
3486 IF (IWK.GT.7) IWK=IWK-7
3487 IF (MO.NE.1) GO TO 50
3488 TDAY=DAY-1
3489 GO TO 80
3490 C
3491 5C DO 60 J=2,12
3492 IF (MO.NE.J) GO TO 60
3493 TDAY=FSTCAY(J-1)+DAY-1
3494 GO TO 70
3495 60 CONTINUE
3496 70 IF (MD.EQ.0.AND.MC.GT.2) TDAY=TDAY+1
3497 80 NTX=TDAY/7
3498 NDX=TDAY-7*NTX+IWK
3499 IF (NDX.GT.7) NDX=NDX-7
3500 WKDAY=NCX
3501 KV=YR/100
3502 KTEST=YR-KV*100
3503 IF (MO.GT.2.OR.KTEST.NE.0) GO TO 90
3504 KV=KV-1
3505 9C LV=KV/4
3506 LTEST=KV-LV*4
3507 IF (LTEST.EQ.2) WKDAY=WKDAY+1
3508 IF (LTEST.EQ.1) WKDAY=WKDAY+2
3509 IF (LTEST.EQ.0) WKDAY=WKDAY+3
3510 WKDAY=WKDAY-3*(LV-4)
3511 100 IF (WKDAY.LE.0) WKDAY=WKDAY+7
3512 IF (WKDAY.LE.0) GO TO 100
3513 IF (WKDAY.GT.7) WKDAY=WKDAY-7
3514 RETURN
3515 C
3516 END
3517 @ELT, IL A41
3518 C THIS IS A MAIN PROGRAM FOR DECODING WEATHER BUREAU TAPES.
3519 DIMENSION DB(24),DP(24),WB(24),WS(24),PB(24),TC(24),TOC(24),PR(24)
3520 2,PS(24)
3521 INTEGER WPOS(10) /13,11,16,19,22,34,43,46,68,70/
3522 INTEGER WLON(10) /3,2,3,3,4,1,1,2,3/
3523 INTEGER TAPE1,TAPE2,DAY,CITY,YEAR,OUT(24),OUTPUT(24,10)
3524 COMMON TAPE1,TAPE2,NO1,NO2,INPUT(1100)
3525 TAPE1=7
3526 TAPE2=8
3527 READ (5,60) NDYSKP,NDAY
3528 NSKIF=4*NDYSKP
3529 CALL NTRAN (TAPE1,7,NSKIF)
3530 NO1=1
3531 NC2=1
3532 DO 50 N=1,NDAY
3533 CALL DECODE (WPOS,WLON,10,OUTPUT,0,YEAR,MONTH,DAY,CITY)
3534 DO 30 K=1,10
3535 DO 10 J=1,24
3536 10 OUT(J)=OUTPUT(J,K)
3537 CALL ERRCR (OUT,K)
3538 DO 20 J=1,24

```

```

3539 20  OUTPLT(J,K)=OUT(J)
3540 30  CONTINUE
3541      DO 40 J=1,24
3542      WS(J)=OUTPUT(J,1)
3543      DB(J)=OUTPUT(J,3)
3544      WB(J)=OUTPUT(J,4)
3545      DP(J)=OUTPUT(J,5)
3546      PB(J)=OUTPUT(J,6)/100.
3547      TC(J)=OUTPUT(J,7)
3548      PR(J)=OUTPUT(J,9)
3549      PS(J)=OUTPUT(J,10)
3550      TO=2
3551      ITK=CUTPUT(J,8)
3552      IF (ITK.EQ.2) TO=1.
3553      IF (ITK.EQ.8.OR.ITK.EQ.9) TO=0.
3554 40   TOC(J)=TC
3555      WRITE (6,80) YEAR,MONTH,DAY,CITY
3556      IF (N.EQ.1) WRITE (6,70) DB,DP,WB,WS,PB,TC,TOC,PR,PS
3557 50   WRITE (8) DB,DP,WB,WS,PB,TC,TOC,PR,PS,YEAR,MONTH,DAY,CITY
3558      END FILE 8
3559      C
3560      C
3561 60   FORMAT (10I7)
3562 70   FORMAT (12F10.3)
3563 80   FORMAT (22H DATA PROCESSED FOR 4I20)
3564      C
3565      END
3565:

```

NBSLD Sample Input/Output

THE TERMINAL AND HAVE YOUR DATA READY ON THE DATA FORMS

RUNID,RUNTP,ASHRAE,IDETAL,METHOD
 RUNID.....IDENTIFICATION OF THE RUN
 1 NEED RESPONSE FACTOR DATA
 2 SKIP RESPONSE FACTOR DATA
 RUNTP.....TYPE OF RUN
 1 ENERGY CALCULATION ..NEEDS WEATHER TAPE
 2 DESIGN LOAD CALCULATION
 3 DESIGN AND ENERGY LOAD CALCULATIONS
 ASHRAE.....0 USE RMTMP
 1 USE ASHRAE WEIGHTING FACTORS
 IDETAL.....0 NO DETAILED OUTPUT
 1 DETAILED OUTPUT
 METHOD.....0 REGULAR TREATMENT FOR THE ROOM
1 SPECIAL TREATMENT OF THE ROOM
 FORT MEYER BLDG 219
 LIGHTING SCHEDULE FOR WEEKDAYS
 EQUIPMENT USAGE SCHEDULE FOR WEEKDAYS
 OCCUPANCY SCHEDULE FOR WEEKDAYS
 LIGHTING SCHEDULE FOR WEEKEND
 EQUIPMENT SCHEDULE FOR WEEKENDS
 OCCUPANCY SCHEDULE FOR WEEKEND
 LIGHTING SCHEDULE FOR THE VACATION PERIOD
 EQUIPMENT USAGE SCHEDULE FOR THE VACATION PERIOD
 OCCUPANCY SCHEDULE FOR THE VACATION PERIOD
 THERMOSTAT SETTING FOR THE COOLING SEASON
 THERMOSTAT SETTING FOR THE HEATING SEASON
 RMOBWO,RMOBBO,RHW,RHS
 DATA SHEET NO 1:NDAY,NSKIP,TAPE2
 DATA SHEET NO 2 +3 :MCNTH,DAY,ELAPS,DBMAX,RANGE,WBMAX, DBMWT,TGS,TGW,UG,LONG,LAT,TZN,ZLF,RHOW
 DATA SHEET NO 4: NAME OF THE ROOM
 DATA SHEET NO 5:IROT,ISKIP,INCLUDE
 DATA SHEET NO 6 N/L,K,P,C,R
 DATA SHEET NO 7: DESCRIPTION OF EACH LAYER

IRF= 1
 WALL COMPOSITION

LAYER	L(I)	K(I)	(I)	C(I)	RES(I)	DESCRIPTION OF LAYERS
1	.031	.120	55.00	.259	.00	GYPSUM BOARD 3/8-IN
2	.000	.000	.00	.000	.20	AIR SPACE 3/4-IN
3	1.080	.580	125.00	.200	.00	COMMON BRICK 13-IN

TIME INCREMENT DT= 1.
 THERMAL CONDUCTANCE U= .431
 RESPONSE FACTORS

J	X	Y	Z
0	1.8144	.0000	4.2967
1	-.6456	.0003	-2.5170
2	-.1642	.0047	-.4141
3	-.1045	.0162	-.2144
4	-.0753	.0271	-.1370
5	-.0581	.0331	-.0974
6	-.0467	.0350	-.0739
7	-.0387	.0341	-.0586
8	-.0326	.0319	-.0480
9	-.0279	.0291	-.0402
10	-.0240	.0262	-.0342
11	-.0208	.0233	-.0293
12	-.0181	.0207	-.0254
13	-.0158	.0182	-.0221
14	-.0138	.0160	-.0192
15	-.0121	.0141	-.0168

COMMON RA_TID CR= .87392

DATA SHEET NO 7: DESCRIPTION OF EACH LAYER

IRF= 2

WALL COMPOSITION

LAYER NO	L(I)	K(I)	(I)	C(I)	RES(I)	DESCRIPTION OF LAYERS
1	.031	.120	55.00	.259	.00	GYPSUM BOARD 3/8-IN
2	.250	.024	3.50	.157	.00	FG WOOL 6-IN

TIME INCREMENT DT= 1.

THERMAL CONDUCTANCE U= .094

RESPONSE FACTORS

J	X	Y	Z
0	.5709	.0650	.1406
1	-.4771	.0286	-.0469
2	-.0001	.0001	-.0000
3	-.0000	.0000	-.0000
4	-.0000	.0000	-.0000

COMMON RATIO CR= .00157

DATA SHEET NO 7: DESCRIPTION OF EACH LAYER

IRF= 3

WALL COMPOSITION

LAYER NO	L(I)	K(I)	(I)	C(I)	RES(I)	DESCRIPTION OF LAYERS
1	.031	.120	55.00	.259	.00	GYPSUM BOARD 3/8-IN
2	.000	.000	.00	.000	1.00	AIR SPACE 4-IN
3	.031	.120	55.00	.259	.00	GYPSUM BOARD 3/8-IN

TIME INCREMENT DT= 1.

THERMAL CONDUCTANCE U= .658

RESPONSE FACTORS

J	X	Y	Z
0	1.0350	.5905	1.0350
1	-.3771	.0674	-.3771
2	.0000	.0000	.0000
3	.0000	.0000	.0000

COMMON RATIO CR= .00000

DATA SHEET NO 7: DESCRIPTION OF EACH LAYER

IRF=

WALL COMPOSITION

LAYER NO	L(I)	K(I)	(I)	C(I)	RES(I)	DESCRIPTION OF LAYERS
1	.010	.122	69.00	.300	.00	ASPHALT TILE 1/8-IN
2	.063	.094	32.00	.200	.00	WOOD FLOOR 3/4-IN
3	.170	.094	32.00	.200	.00	WOOD SUBFLOOR 2-IN

TIME INCREMENT DT= 1.

THERMAL CONDUCTANCE U= .391

RESPONSE FACTORS

J	X	Y	Z
0	1.0303	.1583	.8785
1	-.6150	.2094	-.4640
2	-.0221	.0217	-.0214
3	-.0018	.0018	-.0017
4	-.0001	.0001	-.0001

COMMON RATIO CR= .08120

DATA SHEET NC 7: DESCRIPTION OF EACH LAYER

IRF= 5

WALL COMPOSITION

LAYER NO	L(I)	K(I)	(I)	C(I)	RES(I)	DESCRIPTION OF LAYERS
1	.000	.000	.00	.000	.70	ATTIC SIDE RESISTANCE
2	.073	.094	32.00	.600	.00	WOOD 7/8-IN
3	.000	.000	.00	.000	.08	BUILDING PAPER
4	.050	.750	95.00	.400	.00	GREEN SLATE 1/2-IN

TIME INCREMENT DT= 1.

THERMAL CONDUCTANCE U= .616

RESPONSE FACTORS

J	X	Y	Z
0	.7870	.3149	3.0529
1	-.1623	.2824	-2.3962
2	-.0082	.0177	-.0385
3	-.0004	.0010	-.0021
4	-.0000	.0001	-.0001

COMMON RATIO CR= .05392

DATA SHEET NC 7: DESCRIPTION OF EACH LAYER

IRF= 6

WALL COMPOSITION

LAYER NO	L(I)	K(I)	(I)	C(I)	RES(I)	DESCRIPTION OF LAYERS
1	1.080	.583	125.00	.200	.00	BRICK 13 INCHES

TIME INCREMENT DT= 1.

THERMAL CONDUCTANCE U= .540

RESPONSE FACTORS

J	X	Y	Z
0	4.3078	.0000	4.3078
1	-2.5235	.0015	-2.5235
2	-.4152	.0181	-.4152
3	-.2149	.0457	-.2149
4	-.1373	.0609	-.1373
5	-.0974	.0627	-.0974
6	-.0733	.0576	-.0733
7	-.0573	.0501	-.0573
8	-.0457	.0425	-.0457
9	-.0369	.0355	-.0369
10	-.0300	.0294	-.0300
11	-.0245	.0242	-.0245
12	-.0201	.0200	-.0201
13	-.0165	.0164	-.0165
14	-.0135	.0135	-.0135

COMMON RATIO CR= .81966

DATA SHEET NC 7: DESCRIPTION OF EACH LAYER

IRF= 7

WALL COMPOSITION

LAYER NO	L(I)	K(I)	(I)	C(I)	RES(I)	DESCRIPTION OF LAYERS
1	.250	.024	3.50	.157	.00	FG WOOL 6-INCHES
2	.024	.120	55.00	.259	.00	GYP SUM BOARD 1/4-IN
3	.000	.000	.00	.000	.70	CEILING SIDE RESISTANCE

TIME INCREMENT DT= 1.

THERMAL CONDUCTANCE U= .088

RESPONSE FACTORS

J	X	Y	Z
0	.1393	.0438	.4112
1	-.0505	.0423	-.3121
2	-.0005	.0022	-.0104
3	-.0000	.0001	-.0004
4	-.0000	.0000	-.0000

COMMON RATIO CR= .03509

DATA SHEET NO 8: ROOMNO,QLITY,QEQPY,QCU,FLCG,FRAS,TS,CFMV,ARCHGS,ARCHGW,ARCHGM,ZNORM

DATA SHEET NO.9: IW,IL,TUL,TLL,QCMAX,QHMAX,ITHST,ITK

DATA SHEET NC 10: NS,NW,NN,NE,L,W,H

DATA SHEET 11 AND 12: ROOM SURFACE DATA AND EXTERIOR SURFACE SHADOW

ROOMNO	HT	AG	NOFLR	QCU	ARCHGS	ARCHGW
1.0	10.7	3265.0	1.0	21.0	1.0	1.0

LAT	LONG	TZN	ZNORM
38.0	76.5	5.0	1.0

QLITY	QEQPX	CFMV	DBIN	TG	TV	RHIN
3.0	.2	2000.0	68.0	44.0	65.0	60.0

NEXP	ITK	ITHST
10	0	0

LEND% .4	UCELAG .1	AENDW 47C.0	ATCHT 6.5	IRF	ABSP	U	H	A	WAZ	SHADE	UT	HI
1	1	1	5	.88	.42	6.00	3265.00	.00	.00	.00	.08	.71
2	2	1	1	.70	.32	6.00	370.00	.00	.00	.00	.43	.54
3	3	-1	10	.00	1.06	6.00	4.50	.00	.55	10.89	.54	.54
4	2	1	1	.70	.32	6.00	793.00	90.00	.00	.00	.43	.54
5	3	-1	10	.00	1.06	6.00	155.00	90.00	.55	10.89	.54	.54
6	4	1	10	.90	.61	6.00	50.00	90.00	.00	1.27	.54	.54
7	6	1	3	.00	.45	.00	374.00	180.00	.00	.66	.54	.54
8	2	1	1	.90	.32	6.00	918.00	-90.00	.00	.43	.54	.54
9	3	-1	10	.00	1.06	6.00	80.00	-90.00	.55	10.89	.54	.54
10	8	1	4	.00	.25	1.46	3265.00	.00	.00	.39	.54	.54

74L

SHADOW CASTING DATA

HT	FL	FP	AW	BWL	BWR	D	FP1	A1	B1	C1	FP2	A2	B2	C2
.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0
.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0
.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0
.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0
.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0
.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0
.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0
.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0
.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0
.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0	.0

RADIATION INTERCHANGE FACTORS

SURFACE	1	2	3	4	5	6	7	8	9	10
1	.000	.041	.001	.096	.019	.006	.042	.111	.010	.675
2	.366	.000	.000	.102	.020	.006	.013	.118	.010	.366
3	.366	.000	.000	.102	.020	.006	.013	.118	.010	.366
4	.395	.047	.001	.000	.000	.000	.048	.106	.009	.395
5	.395	.047	.001	.000	.000	.000	.048	.106	.009	.395
6	.395	.047	.001	.000	.000	.000	.048	.106	.009	.395
7	.366	.013	.000	.102	.020	.006	.000	.118	.010	.366
8	.395	.047	.001	.091	.018	.006	.048	.000	.000	.395
9	.395	.047	.001	.091	.018	.006	.048	.000	.000	.395
10	.675	.041	.001	.096	.019	.006	.042	.111	.010	.000

HEATING LOAD IN BTU PER HOUR

SENSIBLE LOAD =	125946.
LATENT LOAD =	6820.
TOTAL LOAD =	132766.

75L

SOLAR DATA (QSUN/OGLOSS)

1	2	3	4	5	6	7	8	9	10
0. 0. 0. 0. 0. 8. 62. 122. 176. 219. 247. 259. 253. 230. 191. 140. 81. 23. 0. 0. 0. 0. 0. 0.									
0. 0.									
0. 0. 0. 0. 0. 3. 14. 39. 71. 98. 116. 123. 120. 105. 81. 50. 18. 7. 0. 0. 0. 0. 0. 0.									
0. 0.									
0. 0. 0. 0. 0. 2. 8. 17. 34. 53. 66. 72. 69. 58. 41. 22. 10. 4. 0. 0. 0. 0. 0. 0.									
0. 0. 0. 0. 0. 3. 12. 19. 24. 28. 30. 33. 77. 124. 159. 175. 160. 86. 0. 0. 0. 0. 0. 0.									
0. 0. 0. 0. 0. 2. 8. 13. 16. 18. 20. 21. 37. 73. 99. 111. 102. 54. 0. 0. 0. 0. 0. 0.									
0. 0. 0. 0. 0. 3. 16. 24. 31. 35. 38. 42. 99. 159. 204. 225. 206. 110. 0. 0. 0. 0. 0. 0.									
0. 0.									
0. 0.									
0. 0. 0. 0. 0. 51. 187. 224. 214. 176. 119. 54. 39. 37. 32. 27. 19. 8. 0. 0. 0. 0. 0. 0. 0.									
0. 0.									
0. 0. 0. 0. 0. 25. 92. 111. 104. 83. 48. 21. 20. 19. 17. 14. 10. 4. 0. 0. 0. 0. 0. 0. 0.									
0. 0.									
0. 0.									

ROOM NAME= OFFICE

DAY	MHR	QLMAX	CLDAY	HLDAY	MCNTH=	DBA
8	21	16	-78842.	-716519.	8	85.5

***** YEAR = 2000 ***** MONTH = 8 ***** DAY = 21

TIME	DBOUT	WBOUT	DBIN	RHIN	QLS	QLL
1	79.4	69.7	84.4	53.6	0.	0.
2	78.4	69.4	83.6	54.8	0.	0.
3	77.6	69.2	82.9	56.1	0.	0.
4	77.0	69.0	82.3	57.2	0.	0.
5	76.8	69.0	81.8	58.2	0.	0.
6	77.2	69.1	81.8	58.2	0.	0.
7	78.2	69.4	82.7	56.5	0.	0.
8	80.0	69.9	78.0	60.0	-36082.	-8260.
9	82.6	70.6	78.0	60.0	-40255.	-7266.
10	85.6	71.5	78.0	60.0	-44928.	-7266.
11	89.0	72.4	78.0	60.0	-49980.	-7266.
12	92.2	73.3	78.0	60.0	-55125.	-7266.

58 0.0,0.4.1
 59
 END PRT

DXCT JINBAR-NESLD

CCNGRATULATIONS## NOW YOU ARE ON NBSLD

WE ASSUME YOU HAVE ALREADY PREPARED THE DATA
 ON NBS DATA FORMS..IF YOU HAD NOT ,PLEASE TURN OFF
 THE TERMINAL AND HAVE YOUR DATA READY ON THE DATA FORMS

RUNID,RUNTYP,ASHRAE,IDETAL,METHOD

RUNID.....IDENTIFICATION OF THE RUN
 1 NEED RESPONSE FACTOR DATA
 2 SKIP RESPONSE FACTOR DATA

RLNTYP.....TYPE OF RUN
 1 ENERGY CALCULATION ..NEEDS WEATHER TAPE
 2 DESIGN LOAD CALCULATION
 3 DESIGN AND ENERGY LOAD CALCULATIONS

ASHRAE.....0 USE RMTMP
 1 USE ASHRAE WEIGHTING FACTORS

IDETAL.....0 NO DETAILED OUTPUT
 1 DETAILED OUTPUT

METHOD.....0 REGULAR TREATMENT FOR THE ROOM
1 SPECIAL TREATMENT OF THE ROOM

FORT MEYER BLDG 219
 LIGHTING SCHEDULE FOR WEEKDAYS
 EQUIPMENT USAGE SCHEDULE FOR WEEKDAYS
 OCCURANCY SCHEDULE FOR WEEKDAYS
 LIGHTING SCHEDULE FOR WEEKEND
 EQUIPMENT SCHEDULE FOR WEEKENDS
 OCCUPANCY SCHEDULE FOR WEEKEND
 LIGHTING SCHEDULE FOR THE VACATION PERIOD
 EQUIPMENT USAGE SCHEDULE FOR THE VACATION PERIOD
 OCCUPANCY SCHEDULE FOR THE VACATION PERIOD
 THERMOSTAT SETTING FOR THE COOLING SEASON
 THERMOSTAT SETTING FOR THE HEATING SEASON

RMCEWD,RMDBSD,RMWRHS
 DATA SHEET NO 1:NCAY,NSKIF,TAPE2
 DATA SHEET NO 2 +3 :MONTH,DAY,ELAPS,DBMAX,RANGE,WBMAX. DBMWT,TGS,TGW,UG,LONG,LAT,TZN,ZLF,RHOW
 DATA SHEET NO 4: NAME OF THE ROOM
 DATA SHEET NO 5: IROT,ISKIP,INCLUDE
 DATA SHEET NO 8: RCMNC,QLITY,GEQPY,QCU,FLCG,FRAS,TS,CFHV,ARCHGS,ARCHGW,ARCHGM,ZNORM
 DATA SHEET NO.9: IW,IL,TUL,TLL,QCMAX,QHMAX,ITHST,ITK
 DATA SHEET NO 10: NS,NW,NN,NE,L,W,H
 DATA SHEET 11 AND 12: ROOM SURFACE DATA AND EXTERIOR SURFACE SHADOW

ROOM NAME= OFFICE		MCNTH=	
DAY	MHR QLMAX	CLDAY	HLDAY DBA
1 1	24 67266.	0.	833917. 33.4
1 2	8 74977.	0.	1557859. 24.0
1 3	8 69210.	0.	1339045. 28.8
1 4	8 51673.	0.	767721. 41.8
1 5	24 46525.	0.	829164. 36.1
1 6	6 54181.	0.	1099831. 31.3
1 7	8 49389.	0.	982197. 35.5
1 8	4 43912.	0.	923993. 35.5
1 9	4 44824.	0.	851840. 36.7
1 10	24 49385.	0.	871738. 37.6
1 11	9 64347.	0.	1239178. 28.5
1 12	4 54753.	0.	1079643. 32.6
1 13	24 46914.	0.	855627. 37.5
1 14	24 71760.	0.	1394308. 24.4
1 15	17 72942.	0.	1742967. 17.4
1 16	7 72919.	0.	1588378. 22.8
1 17	15 74586.	0.	1768679. 13.8
1 18	11 73647.	0.	1680665. 18.2
1 19	4 74156.	0.	1430075. 25.0
1 20	5 63771.	0.	1141373. 32.5
1 21	8 46194.	0.	765276. 40.3
1 22	5 31852.	0.	436868. 48.6
1 23	24 48898.	0.	335641. 49.1
1 24	8 68003.	0.	1301764. 25.4
1 25	6 5674.1	0.	1157575. 30.0
1 26	6 48566.	0.	826622. 38.5
1 27	1 37008.	0.	707440. 40.8
1 28	8 40155.	0.	797525. 37.2
1 29	8 43283.	0.	789407. 38.5
1 30	8 42105.	0.	789377. 38.0
1 31	8 43971.	0.	875430. 35.2

MCNTHLY COOLING LOAD= .0000000 BTU

MCNTHLY HEATING LOAD= .32761124+08 BTU

MAX COOLING LOAD = 0. MCNTH = 0 DAY = 0 HOUR = 0
 MAX HEATING LOAD = 74977. MCNTH = 1 DAY = 2 HOUR = 8
 TCTAL COOLING CONSUMPTION FOR 1 ROOMS = .00000 BTU
 TCTAL HEATING CONSUMPTION FOR 1 ROOMS = .32761+08 BTU
 DATA SHEET NO 4: NAME OF THE ROOM

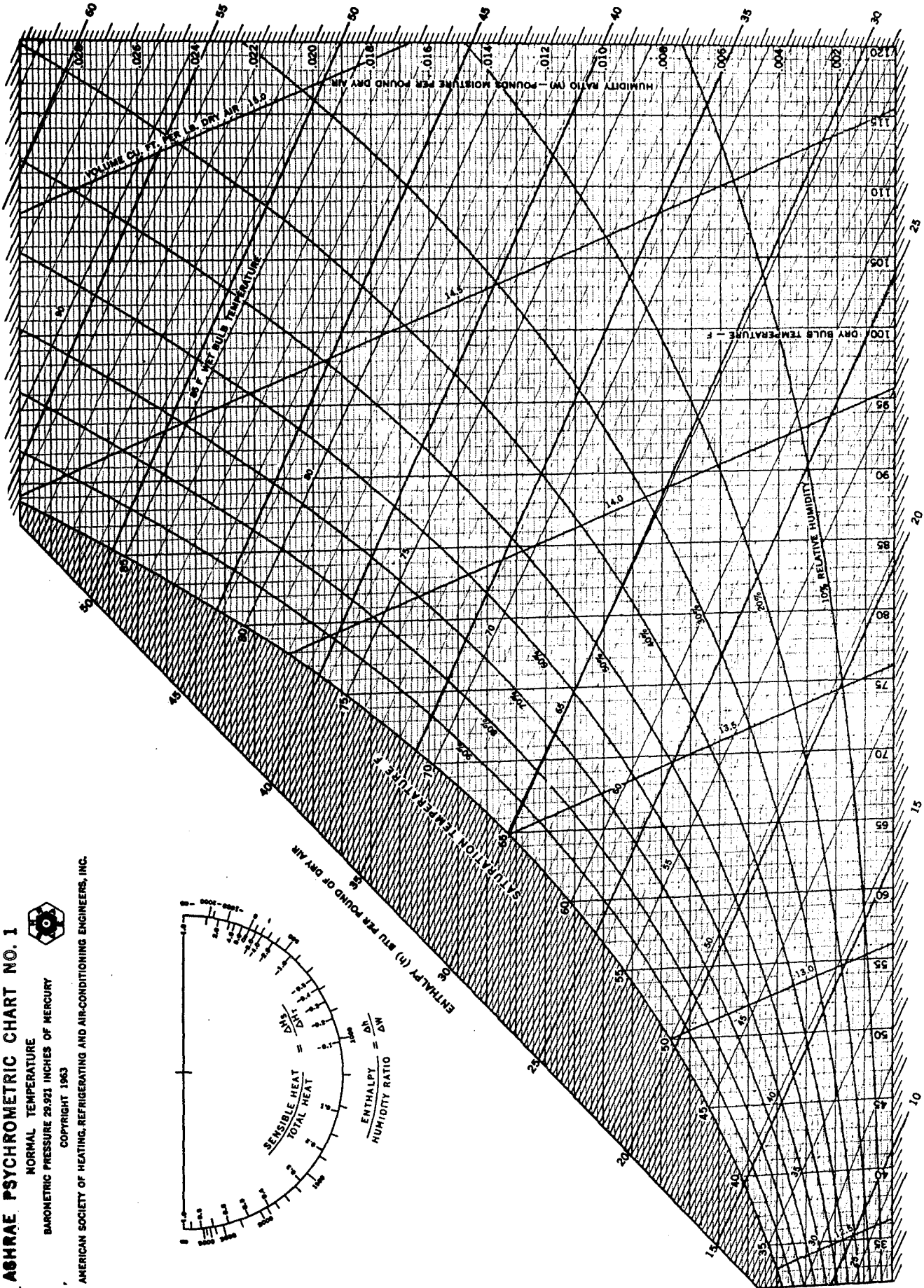
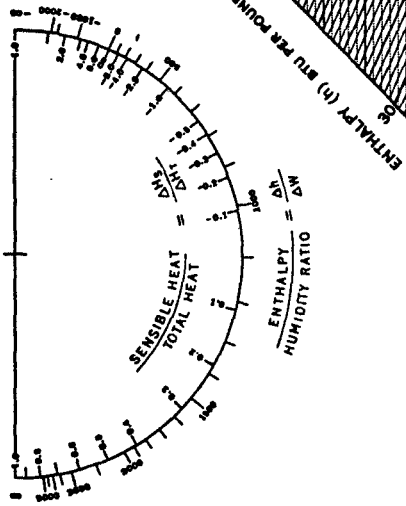
DBRKT PRINTS

11. ASHRAE PSYCHROMETRIC CHART NO. 1

NORMAL TEMPERATURE
 BAROMETRIC PRESSURE 29.921 INCHES OF MERCURY
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AMERICAN SOCIETY OF HEATING, REFRIGERATING AND AIR-CONDITIONING ENGINEERS, INC.



ENTHALPY (h) BTU PER POUND OF DRY AIR