

(NASA-CR-3169) A SIMPLIFIED COMPUTER  
PROGRAM FOR THE PREDICTION OF THE LINEAR  
STABILITY BEHAVIOR OF LIQUID PROPELLANT  
COMBUSTORS (Colorado State Univ.) 59 p  
HC A04/HF A01

N79-28226

Unclas  
CSCL 21H H1/20 34097

**NASA Contractor Report 3169**

**A Simplified Computer Program  
for the Prediction of the  
Linear Stability Behavior of  
Liquid Propellant Combustors**

**C. E. Mitchell and K. Eckert**  
*Colorado State University*  
*Fort Collins, Colorado*

**Prepared for**  
**Lewis Research Center**  
**under Grant NGR-06-002-095**



**National Aeronautics  
and Space Administration**

**Scientific and Technical  
Information Branch**

1979

## TABLE OF CONTENTS

NOMENCLATURE . . . . .	v
INTRODUCTION . . . . .	1
THEORY . . . . .	3
Combustor Model . . . . .	3
Method of Solution . . . . .	10
COMPUTATIONAL METHODS. . . . .	15
Matrix Sizing and Program Convergence . . . . .	15
Program Options . . . . .	18
Choice of Fundamental Acoustic Mode . . . . .	21
REFERENCES . . . . .	23
Program MODULE . . . . .	25
General Description of Program . . . . .	25
Program Input . . . . .	32
Program Output . . . . .	34
Sample Run . . . . .	34
Program MODULE Listing . . . . .	40

## NOMENCLATURE

### Letters

- a - speed of sound
- AMF - aperture mean flow
- BR - backing distance for slot absorber
- F - quantity for integral governing equation
- f - quantity for integral governing equation
- $G_N$  - modified Green's function, defined after Equation 15
- $i$  -  $\sqrt{-1}$
- $J_m$  - Bessel Function of first kind of order  $m$
- K - acoustic impedance of slot absorber
- L - nondimensional chamber length
- $\ell$  - radial acoustic mode assumed
- $L_a$  - actual length of aperture for slot absorber
- $L_{eff}$  - effective length of aperture for slot absorber
- M - mean flow Mach number
- $m$  - transverse acoustic mode assumed
- $\dot{m}$  - mass generation rate
- $n$  - longitudinal acoustic mode assumed
- $n$  - interaction index
- $\vec{n}$  - outward directed normal unit vector
- P - pressure
- r - radial dimension
- $r_c$  - radius of chamber
- $R_0$  - resistance of slot absorber
- S - surface of combustion chamber over which integration is to be carried out

- T - temperature
- t - time
- $u_{\theta}$  - normal component of velocity oscillation in tangential direction
- $u_r$  - normal component of velocity oscillation in radial direction
- $u_t$  - transverse velocity
- V - velocity or volume of combustion chamber
- Wa - aperture width for slot absorber or length of acoustic liner
- $\chi_a$  - distance from injector face to beginning of slot absorber or acoustic liner
- $\chi_b$  - distance from injector face to end of slot absorber or acoustic liner
- z - longitudinal dimension

Greek Letters

- $\beta$  - acoustic admittance of a surface
- $\gamma$  - ratio of specific heats
- $\varepsilon$  - nondimensional wave amplitude
- $\eta$  - acoustic eigenvalue
- $\theta$  - angle in radians
- $\Lambda$  - normalization factor defined after Equation 15
- $\lambda_{\ell m}$  - root of Bessel Function of first kind, such that  $J'_m(\lambda_{\ell m}) = 0$
- $\mu$  - coefficient matrix
- $\rho$  - density
- $\tau$  - sensitive time lag
- $\phi$  - velocity potential

$\psi$  - normalization factor defined after Equation 15

$\Omega_{lmn}$  - acoustic eigenfunction

$\omega$  - complex frequency

### Superscripts

$\rightarrow$  - vector quantities

$*$  - dimensional quantity

' - derivative with respect to argument, or perturbation quantity

- - mean or steady state quantity

### Subscripts

I - injector

L - liner

N - nozzle

## INTRODUCTION

The purpose of this report is to present an analytical technique and a computer program which can be used for the prediction of the linear stability behavior of liquid propellant combustors. The technique involved has been developed over the last few years at Colorado State University in the examination of several aspects of the instability problem. Basically, the approach employs a Green's function integral method in the iterative determination of combustor frequency, decay rate and spatial waveform.

This general approach has been applied to several different combustor models in the examination of different aspects of the linear instability problem. (Ref. 1-10.) This work was performed by several different people (mainly graduate students), and a wide variety of nomenclature and programming techniques has resulted. The details of the analytical approach have also varied from author to author though the general method remained the same. This more or less comprehensive compendium of programs and analyses as it exists in its several forms is cumbersome, somewhat redundant, and certainly hard to use as a designer's tool.

With this in mind it was decided to develop a simplified stability analysis and computer program which contained the most important features of the earlier work in a format that would be relatively easy to use. Consequently, the main goal of this effort has been the development of a computer program simple enough to be used effectively by a person without an exhaustive background in either advanced mathematics or stability theory.

In order to do this some compromises have had to be made as far as comprehensiveness and accuracy are concerned, and some aspects of the stability

problem treated previously have not been included. For example, the effect of distributing combustion sources along the combustor axis (as opposed to having a concentrated combustion zone near the injector) on overall stability has been studied and analyzed using two different approaches (Ref. 8, 9). This effect is not included in the simplified model presented here, however. The justification for this is based on the fact that much greater complexity is introduced into both the analysis and the computer program when distributed sources of combustion are considered, while the qualitative stability behavior is very similar to that predicted for concentrated combustion. Moreover, the quantitative effect of distributing the combustion is stabilizing relative to the predictions for a concentrated combustion zone. Thus, the simplified model presented here will tend to give conservative estimates of combustor stability when the combustor being examined has its combustion zone well distributed (axially).

Other effects such as irrotationality, entropy variations, and droplet drag effects have also been ignored since their influence has been found to be small, stabilizing or both.

The body of the report is divided into three main sections. The first (called "Theory") presents the model and method of analysis. The second section (called "Computational Methods") presents the basics of the computational method and the user options available. The final section (called "Program MODULE") gives a user's manual, sample input and output and a flow chart. It is not necessary for a person wishing to use the computer program (MODULE), to follow the analytical details of the first section. It will be necessary, however, for him to understand the basics of the model and general method of approach as presented in that section so that appropriate input to the program may be made and correct interpretation of the output can result.



## THEORY

### Combustor Model

The motor configuration considered here is characterized by circular cylindrical geometry, a concentrated combustion zone located at the injector end of the combustor, a nozzle at the opposite end, and either an acoustic liner or a slot absorber located in the cylindrical walls. A sketch of the combustor model is given in Figure 1. In the development of a linear stability model for a combustor of this type it is first necessary to represent the four main features of the configuration using appropriate mathematical models. The four aspects of the problem requiring such modeling are

- 1) The gasdynamic flow field
- 2) The combustion zone
- 3) The nozzle
- 4) The acoustic liner or absorber.

Each of these will be discussed separately before going on to a presentation of the global stability model and analytical technique.

#### 1) The gasdynamic flow field

The flow field downstream of the concentrated combustion zone is taken to consist of a single component, single phase product gas which is non-conducting, inviscid and calorically perfect. The flow is assumed to be homentropic and irrotational. As long as the combustion zone is concentrated and pressure waves are of small amplitude, it has been shown that these approximations are not severely limiting and self-consistent (Ref. 11, 12). Before presenting the equations describing this flow field the relevant state and flow variables are non-dimensionalized as follows.

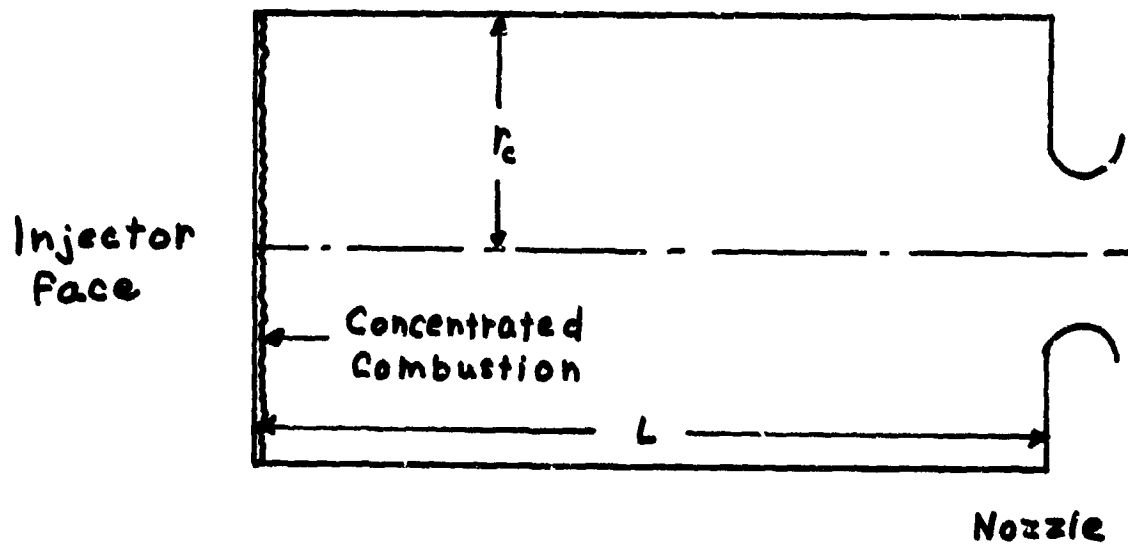


Figure 1. Combustion Chamber

$$\rho = \rho^*/\bar{\rho}^* \quad , \quad T = T^*/\bar{T}^*$$

$$\vec{V} = \vec{V}^*/\bar{a}^* \quad , \quad P = P^*/\bar{P}^*$$

The independent variables are nondimensionalized as follows.

$$t = t^*(r_c^*/\bar{a}^*) \quad , \quad r = r^*/r_c^*$$

$$z = z^*/r_c^*$$

where \* denotes dimensional quantities and  $\bar{\quad}$  denotes mean chamber values.

Using this nondimensional scheme the conservation equations become

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{V}) = 0 \quad \text{CONTINUITY} \quad (1)$$

$$\rho \frac{D\vec{V}}{Dt} + \frac{1}{\gamma} \nabla P = 0 \quad \text{MOMENTUM} \quad (2)$$

$$P = \rho^\gamma \quad \text{HOMENTROPIC} \quad (3)$$

$$P = \rho T \quad \text{STATE} \quad (4)$$

$$\vec{V} = \nabla \phi \quad \text{IRROTATIONALITY} \quad (5)$$

Under the assumption of small amplitude oscillations the state and flow variables are represented as the sum of a mean (steady state) component and an oscillatory component, products of which are ignored as being higher order terms. Thus

$$P = 1 + p' \quad \phi = Mz + \phi$$

$$\rho = 1 + \rho'$$

$$T = 1 + T' \quad \vec{V} = M\vec{e}_z + \nabla \phi$$

where  $\hat{e}_z$  is the unit vector in the axial direction and  $M$  is the mean flow Mach number.

After some manipulation the conservation equations can be reduced to a simple scalar partial differential equation

$$\nabla^2 \phi - \frac{\partial^2 \phi}{\partial t^2} = 2M \frac{\partial^2 \phi}{\partial t \partial z} + M^2 \frac{\partial^2 \phi}{\partial z^2} \quad (6)$$

The equation relating the state variables to  $\phi$  is

$$p' = -\gamma \left( \frac{\partial \phi}{\partial t} + M \frac{\partial \phi}{\partial z} \right) \quad (7)$$

Periodic oscillations in time are assumed so that

$$p' = p(r, \theta, z) e^{i\omega t}$$
$$\phi = \phi(r, \theta, z) e^{i\omega t}$$

where  $\omega = \omega_R + i\lambda$  is the complex frequency,  $\omega_R$  the frequency,  $\lambda$  the decay rate. (If  $\lambda > 0$  decay occurs.)

## 2) Combustion zone response model

It is assumed that all combustion occurs in a length small compared with the combustor's axial dimension. In the steady state mass is produced at the rate  $\bar{m} = M$  in the nondimensional system used here. No attempt to describe the details of the combustion process is made. Instead it is simply assumed that the combustion zone is sensitive to pressure oscillations and responds to these oscillations through a combustion zone admittance function  $\beta_I$ .

Thus

$$\vec{\nabla} \phi \cdot \vec{n} = \beta_I p' \quad (z = 0) \quad (8)$$

$\beta_I$  is taken to be a constant for the entire combustion zone, though  $p'$  is, of course, a function of  $r$  and  $z$  as well as time. In terms of the mass perturbation rate,  $\dot{m}'$ , the response condition is

$$\dot{m}' = \left( \frac{M}{\gamma} - \beta_I \right) p' \quad (9)$$

$\beta_I$  is, in general, complex so that all phasings between  $\dot{m}'$  (or  $u'$ ) and  $p'$  are possible. Note that if the real part of  $\beta_I$  is greater than  $\frac{M}{\gamma}$ , the combustion zone provides a damping rather than driving effect.

It is also possible to relate  $\beta_I$  to the interaction index  $n$ , and time lag  $\tau$  of the Crocco sensitive time lag model. The appropriate relationship is

$$\beta_I = M \left( \frac{1}{\gamma} - n (1 - e^{-i\omega\tau}) \right) . \quad (10)$$

Values of  $\beta_I$  (or  $n$ , and  $\tau$ ) must be supplied by the program user or calculated as output, given all other parameters. These options will be discussed later.

### 3) Nozzle model

Here again no attempt is made to investigate the details of the nozzle flow and, instead, a nozzle admittance function  $\beta_N$  is used.

$$\vec{\nabla}\phi \cdot \vec{n} = \beta_N p' \quad (z = L) \quad (11)$$

Values of  $\beta_N$  are to be supplied by the user. Tables of admittance functions are given in Ref. (13), for example, for conical nozzles. In the absence of any knowledge of the nozzle response value it is suggested that the simple "short" nozzle value

$$\beta_N = M \left( \frac{\gamma-1}{2\gamma} \right)$$

be used.

4) Acoustic liner or slot absorber model.

Two possibilities are considered. The first is an acoustic liner of uniform average admittance,  $\beta_L$ , which is uniform in the azimuthal ( $\theta$ ) direction and extends along the cylindrical wall from  $z = x_A$  to  $z = x_B$ . For this liner the appropriate boundary condition is

$$\vec{\nabla}\phi \cdot \vec{n} = \beta_L p' \quad (12)$$

No attempt is made to calculate  $\beta_L$  in either the analysis or computer program and therefore  $\beta_L$  must be supplied by the user.

The second absorber configuration considered is a circumferential slot machined into the cylindrical wall of the chamber and acting as a Helmholtz resonator. The geometry assumed is shown in Figure 2. All dimensions are nondimensional through division with the chamber radius.

The appropriate boundary condition at  $r = 1$  (chamber wall) over the aperture width  $W_A (= [x_B - x_A])$

$$\vec{\nabla}\phi \cdot \vec{n} = \beta_L p$$

$$\text{or } \vec{\nabla}\phi \cdot \vec{n} = \frac{1}{\gamma K} p$$

where  $K = \frac{1}{\gamma\beta_L}$  is the impedance at the aperture entrance.  $K$  is used in this case to be consistent with existing treatments of Helmholtz resonators of this general type.  $\beta_L$  and  $K$  are, in general, complex with  $K = R_0 + ik$ , where  $R_0$  is the resistance,  $k$  the reactance. Standard relationships for  $R_0$ , resistance and  $L_{\text{eff}}$  taken from Reference (15) and (16) respectively, are given below.

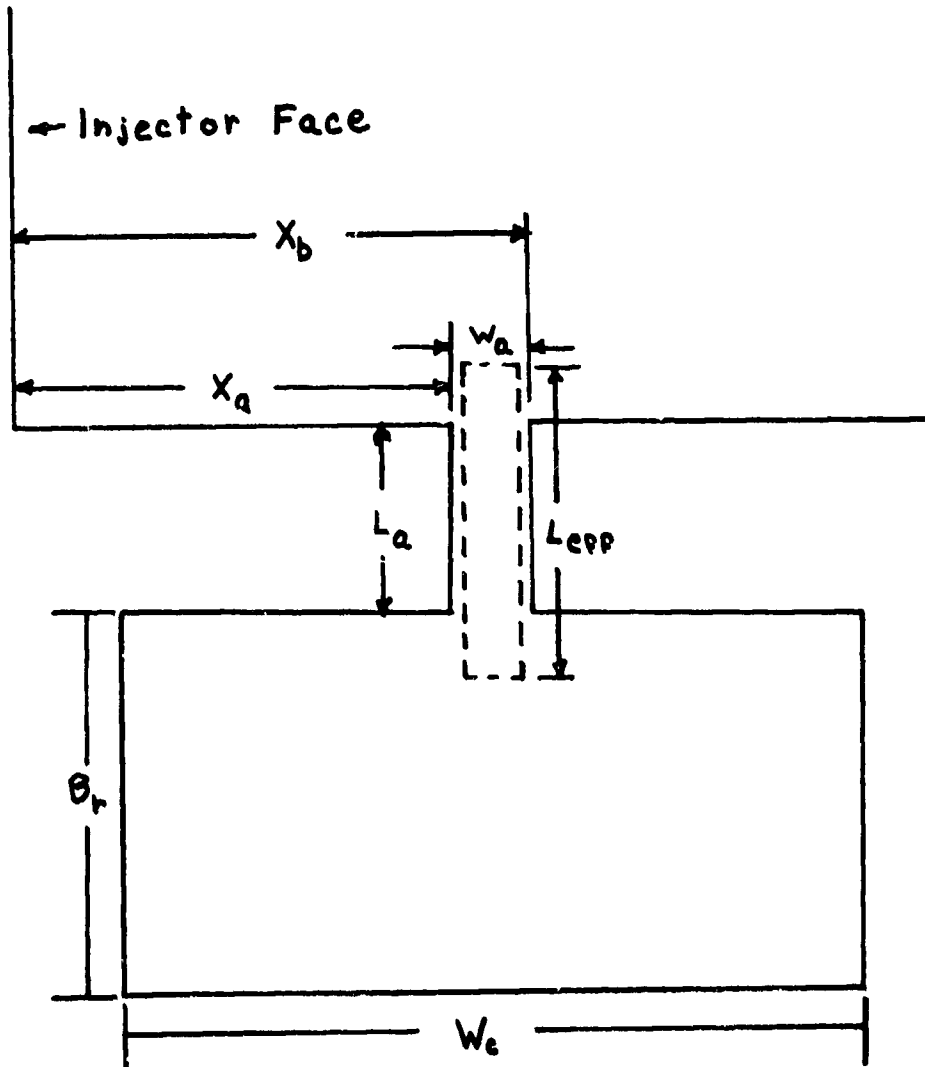


Figure 2. Slot Absorber Geometry

$$R_0 = \left[ .8 \left( 1.5(\text{AMF})R_0 + \frac{\epsilon |p|}{\gamma \left( 1 + \frac{k^2}{R_0^2} \right)^{1/2}} \right) \right]^{1/2} \quad (15)$$

where  $\epsilon$  is wave amplitude and  $p$  is the modulus of the pressure at  $r = 1$ .

$$L_{\text{eff}} = L_A + (0.375)(0.85)W_A \left[ 1 - 0.7 \left( \frac{W_A}{W_C} \right)^{1/2} \right] \quad (5)$$

An expression for the reactance  $k$ , comes directly from linear Helmholtz resonator theory and is given by

$$k = \omega_R \bar{\rho}_{\text{ap}} L_{\text{eff}} - \frac{W_A \bar{a}_a^2 \bar{\rho}_a}{\omega_R W_C BR}$$

where  $\bar{\rho}_{\text{ap}} = \left( \frac{\bar{\rho}_a^*}{\bar{\rho}^*} \right)$ ,  $\bar{a}_a^2 = \left( \frac{\bar{a}_a^*}{\bar{a}^*} \right)$ ,  $\bar{\rho}_a = \left( \frac{\bar{\rho}_a^*}{\bar{\rho}^*} \right)$  are, respectively,

the nondimensional aperture density, absorber cavity mean sound speed, and absorber cavity density, and  $\omega_R$  is the real part of the oscillation frequency. All these quantities, as well as the geometrical quantities in Fig. 2, AMF, and the assumed wave amplitude,  $\epsilon$ , must be supplied by the user. The expression for  $R_0$ , Equation (4), is then solved iteratively for  $R_0$  as a function of frequency. Thus, an expression for  $K(\omega)$  (or  $\beta_L(\omega)$ ) is found numerically.

#### Method of Solution

The governing partial differential equation, Equation (6) along with the necessary boundary conditions (Equations (8) (10) (12) or (13)) are transformed to integral form using a Green's function, and the resulting integral equations are solved iteratively. Details of the transformation and solution method are presented in References (1, 2, 7). Only those relationships, definitions, and



equations necessary for understanding and using the computer program which determines combustor stability will be presented here.

The transformed integral equations for  $\phi(r, \theta, z)$  and  $\omega$  are

$$\begin{aligned} \phi = \Omega_{\hat{\ell}\hat{m}\hat{n}} + \iiint_{V_0} G_N(\vec{r}/\vec{r}_0) F_1(\phi) dV_0 \\ + \iint_{S_0} G_N(\vec{r}/\vec{r}_0) f_1(\phi) dS_0 \end{aligned} \quad (14)$$

$$\omega^2 - \eta_{\hat{\ell}\hat{m}\hat{n}}^2 = \iiint_V \Omega_{\hat{\ell}\hat{m}\hat{n}} F_1(\phi) dV + \iint_S \Omega_{\hat{\ell}\hat{m}\hat{n}} f_1(\phi) dS \quad (15)$$

where  $F_1(\phi) = 2i\omega M \frac{\partial \phi}{\partial z} + M^2 \frac{\partial^2 \phi}{\partial z^2}$

$$f_1 = -\beta p$$

$$\beta = \beta_N \text{ at nozzle } (z = L)$$

$$\beta = \beta_I \text{ at combustion zone } (z = 0)$$

$$\beta = \beta_L \text{ (or } \frac{1}{\gamma K} \text{) at liner (or absorber) } (r = 1)$$

$$\beta = 0 \text{ on all other surfaces}$$

$$G_N(\vec{r}/\vec{r}_0) = \sum_{\hat{\ell}} \sum_{\hat{m}} \sum_{\hat{n}} \frac{\Omega_{\hat{\ell}\hat{m}\hat{n}}(\vec{r}) \Omega_{\hat{\ell}\hat{m}\hat{n}}(\vec{r}_0)}{(\omega^2 - \eta_{\hat{\ell}\hat{m}\hat{n}}^2)}$$

$$\hat{\ell} \neq \hat{\ell}, \hat{m} \neq \hat{m}, \hat{n} \neq \hat{n}, \text{ simultaneously}$$

$$\Omega_{\ell mn} = \frac{J_m(\lambda_{\ell m} r) \cos \frac{n\pi z}{L} \cos m \theta}{\Lambda_{\ell mn}^{1/2}}$$

$$\Lambda_{\ell mn} = \iiint_V \left[ J_m(\lambda_{\ell m} r) \cos \frac{n\pi z}{L} \cos m \theta \right]^2 dV$$

$\lambda_{\ell m}$  are the roots of  $J'_m(\lambda_{\ell m}) = 0$

$$\eta_{\ell mn}^2 = \lambda_{\ell m}^2 + \left( \frac{n\pi}{L} \right)^2$$

$\Omega_{\ell mn}$  are the normalized eigenfunctions for a cylindrical chamber with no mean flow and non reactive walls.  $\ell$ ,  $m$ ,  $n$  are the set of integers giving the radial, azimuthal, and axial character of the particular eigenfunction (or acoustic mode) in question. Thus,  $\Omega_{110}$  represents a first transverse mode,  $\Omega_{120}$  a second transverse mode,  $\Omega_{200}$  a first radial mode,  $\Omega_{001}$  a first axial mode,  $\Omega_{111}$  a combined first transverse first axial mode, etc. The associated eigenvalues (acoustic frequencies) are

$$\eta_{\ell mn}^2 = \lambda_{\ell m}^2 + \left( \frac{n\pi}{L} \right)^2 .$$

The solution technique revolves around the assumption that the actual solution including mean flow and reactive walls has a character that is reasonably close to one of these acoustic modes. The particular acoustic mode most characteristic of the overall oscillation is called  $\Omega_{\hat{\ell}\hat{m}\hat{n}}$ , where  $\hat{\ell}$ ,  $\hat{m}$ ,  $\hat{n}$  are the associated indices giving the radial, azimuthal, and axial character. The related eigenvalue (acoustic frequency) is  $\eta_{\hat{\ell}\hat{m}\hat{n}}$ . A discussion of the selection of  $\Omega_{\hat{\ell}\hat{m}\hat{n}}$  in applications will be given later.

The equation for  $\phi$ , Equation (14), implies that  $\phi$  takes the following form

$$\phi = \sum_{\ell} \sum_{n} \sum_{m} \mu_{\ell mn} J_m(\lambda_{\ell m} r) \cos \frac{n\pi z}{L} \cos m \theta$$

where the coefficient matrix  $\mu_{\ell mn}$  is determined by evaluation of the integrals on the right hand side of Equation (14).

Because of the symmetry in the  $\theta$  direction which results from the assumptions concerning the boundary conditions, the series in  $m$  actually contains only one term,  $\hat{m}$ .

Thus, for the model used here  $\phi$  may be written

$$\phi = \sum_{\ell} \sum_{n} \mu_{\ell n} J_{\hat{m}}(\lambda_{\ell \hat{m}} r) \cos \frac{n\pi z}{L} \frac{\cos \hat{m} \theta}{\epsilon_{\theta}} \quad (16)$$

where  $\epsilon_{\theta}^2 = \int_0^{2\pi} (\cos \hat{m} \theta)^2 d\theta$ . Exactly the same coefficient matrix would result if traveling waveforms were assumed. In this case

$$\phi = \phi(r, z) e^{i(\omega t + \hat{m}\theta)}$$

$$\phi = \sum_{\ell} \sum_{n} \mu_{\ell n} J_{\hat{m}}(\lambda_{\ell \hat{m}} r) \cos \frac{n\pi z}{L} \quad (17)$$

and  $\mu_{\ell n}$  would be identical to the standing wave matrix.

The matrix  $\mu_{\ell n}$  and the complex frequency  $\omega$  are determined by an iterative process. The lowest order guess for  $\phi$  (or  $\mu_{\ell n}$ ) is used in the integral expressions of Equations (14) and (15) to compute improved values for the  $\mu_{\ell n}$  and  $\omega$ . The process continues until successive iterations are invariant to some degree of accuracy. A natural choice for the lowest order estimate for  $\phi$  would be  $\Omega_{\hat{\ell}\hat{m}\hat{n}}$ ; the lowest order frequency would then be  $\eta_{\hat{\ell}\hat{m}\hat{n}}$ . Though these initial guesses will work in general,

experience has indicated that convergence can be slow and matrix sizes large, particularly when the mean flow Mach number is greater than about 0.3. Better convergence and a smaller matrix size are possible if the separation of variables solution for a combustor with mean flow but without an absorber is used. This solution was originally developed by Priem and Rice (Ref. (14)); in the modified form appropriate here it is discussed in References (1) and (2). The computer program presented later uses this form as the lowest order  $\phi$ .

In addition to assuming a lowest order form for  $\phi$  and  $\omega$  and iterating, it is also possible to fix  $\omega$  at some prescribed value (supplied by the user) and iterate to find the appropriate  $\mu_{2n}$  and  $\beta_1$  (or  $n$ , and  $\tau$ ) from the same equations. The latter approach is used to solve for the combustion response necessary to sustain an oscillation of a given frequency and decay (growth) rate and known absorber and nozzle admittances. It would also be possible to set up the technique to solve iteratively for another parameter, such as nozzle admittance, for given combustion admittance; however, this has not been done in the program presented here.

### COMPUTATIONAL METHODS

As discussed in the "Theory" section, Equations (14) and (15) are set up for iterative solution. Computer program MODULE is an algorithm for performing the necessary iterative computations on a digital computer. Several different choices are possible as far as input, output, and accuracy are concerned. These choices will be discussed in this section.

#### Matrix Sizing and Program Convergence

In the solution of Equations (14) and (15) two variables are always iterated. One of these is the perturbation velocity potential  $\phi(r, \theta, z)$ . The other is either the complex frequency  $\omega$  ( $\omega_R + i\lambda$ ) or the complex combustion admittance,  $\beta_I$  (real ( $\beta_I$ ) + i imag ( $\beta_I$ )).

The perturbation velocity potential is represented by a series expansion (Equation (16)).

$$\phi(r, \theta, z) = \left[ \sum_{\ell n} \mu_{\ell n} J_{\hat{m}}(\lambda_{\ell \hat{m}} r) \cos \frac{n\pi z}{L} \right] \frac{\cos \hat{m} \theta}{\varepsilon_{\theta}}$$

Thus, solution for the coefficient matrix  $\mu_{\ell n}$  yields  $\phi(r, \theta, z)$  and, in fact, it is this matrix which is the actual iterated variable in the solution algorithm. Formally,  $\mu_{\ell n}$  is doubly infinite in  $\ell$  and  $n$ . That is,  $1 \leq \ell < \infty$ ,  $0 \leq n < \infty$ . As a practical matter, however, limits on the largest values  $\ell$  and  $n$  may take (in other words the dimensions of matrix  $\mu_{\ell n}$ ) must be determined. It should be recalled here that the integers  $n$  are associated with axial dependence (through  $\cos \frac{n\pi z}{L}$ ) while the integers  $\ell$  are associated with radial dependence (through  $J_{\hat{m}}(\lambda_{\ell \hat{m}} r)$ ).

Any choice for the maximum number of " $\ell$  terms" and " $n$  terms" will limit accuracy. A compromise between program run time, storage requirements,

and accuracy is desirable. Naturally, no one choice will be optimal for all combustor configurations. However, hundreds of runs with "typical" designs have indicated some rules of thumb to be used.

First of all, in none of the combustors investigated was any significant increase in accuracy obtained by keeping more than 50 terms in the axial direction or ten terms in the radial direction. That is, keeping 100 terms in the axial direction or 20 terms in the radial direction affected the values of the iterated variables only very slightly ( $<0.25\%$ ). Consequently, the program as written accepts a  $10 \times 50$  matrix size for  $\mu_{\ell n}$  as the maximum allowable. In the program variables this means  $LTS \leq 10$ ,  $NTS \leq 50$ , where LTS and NTS are, respectively, the number of terms in the " $\ell$ " direction and the number of terms in the " $n$ " direction.

The question as to the "best" values of LTS and NTS to use in a given combustor configuration is difficult to answer. Eckert (Ref. (14)) has studied optimal values for LTS and NTS for a "typical" configuration and suggests values of 3 for LTS and 16 for NTS. However, for a combustor with no absorber, a single term ( $\hat{\ell}$ ) is necessary for description of the radial field and  $LTS = 1$  in this case. On the other hand, if the Mach number is small, mean flow effects are less important and fewer terms in the axial direction (smaller NTS) would be needed. However, for configurations with large absorber effects or high Mach numbers ( $> 0.4$ ) it is likely that "best" values for LTS and NTS could be greater than 3 and 16, respectively.

With this in mind it is suggested that the values  $LTS = 3$  and  $NTS = 16$  be used as a general rule. If strong absorber or high Mach number effects are present and may compromise accuracy, it is suggested that results with  $LTS = 9$  and  $NTS = 50$  be computed and compared with the smaller matrix results to estimate accuracy. Values of LTS and NTS larger than 3 and 16

could then be inserted until the desired accuracy relative to the  $9 \times 50$  size was obtained. It should be noted that improvement in accuracy is monotonic with increasing NTS. The same is not true for LTS because of the alternating nature of the series involved and best results occur if LTS is an odd number (3, 5, 7, 9).

Once the dimensions of the  $\mu_{\lambda n}$  matrix are determined it is next necessary to decide upon an acceptable convergence condition for the iteration process. The second iterated variable (either  $\omega$  or  $\beta_I$  depending upon the application) is used to do this. Successive values of the iterated variable are compared. When the difference between the two values is less than some value, adequate convergence is assumed. Since both  $\omega$  and  $\beta_I$  are complex numbers, it is necessary that both the real and imaginary parts converge in the sense just mentioned. In this program, however, it is convenient instead to deal with  $\omega$  (or  $\beta_I$ ) in complex polar notation, and require that successive values of the modulus and phase angle converge. This is because the phase angle is frequently near zero and can cause problems in the definition of convergence for the imaginary part of the iterated variable. In program MODULE convergence is assumed when the percent change in the modulus of the iterated variable is less than the value ERROR and, at the same time, the absolute value of the change in the phase angle is also less than ERROR. ERROR can take values between  $10^{-5}$  and 1.0.

In most cases convergence to within 0.1% or less is rapid, usually occurring in ten iterations or less. However, for some choices of parameters and for some program options it can be much slower or not occur at all. For this reason a maximum desired number of iterations must be specified. This is done through program variable IDMAX which can take any integer value. If convergence does not occur in the number of iterations

specified by IDMAX, the iterative loop terminates, and program values at the last iteration are output.

### Program Options

In addition to choosing either  $\omega$  or  $\beta_I$  as the iterated variable, choices are possible as far as the form of the combustion response model and the acoustic absorber. Taken together this results in six distinct ways of running the program. These possibilities are labelled options and are described sequentially below. For all of the options it is necessary that certain design or program variables be specified by the user. These parameters are  $\gamma$  (ratio of specific heats),  $M$  (mean flow Mach number),  $L$  (chamber length to radius ratio),  $\beta_N$  (complex nozzle admittance), ERROR (maximum error allowable in determining convergence), LTS (number of terms in radial direction kept), and NTS (number of terms kept in the axial direction).

### Option 1

This option is designed to compute frequency and decay rate (complex frequency) for known combustion zone admittance and known acoustic absorber (or liner) length and admittance. The iterated variable is the complex frequency. Required to be input to the program are  $\beta_I$ ,  $\beta_L$ ,  $X_A$  and  $X_B$ .  $X_A$  and  $X_B$  are the nondimensional distances to the start and end of the acoustic absorber, respectively. Output are  $\omega_R$  and  $\lambda$ ,  $\mu_{ln}$ , the input parameters, and  $n$  and  $\tau$ , the interaction index and time lag corresponding to the given  $\beta_I$  and the converged value for  $\omega$ .



### Option 2

Option 2 is similar to Option 1 except that the combustion response is described by  $n$  and  $\tau$  instead of  $\beta_I$ . In this case  $n$  and  $\tau$  are input and  $\beta_I$  is output. Other input and output parameters are the same as for Option 1.

### Option 3

In this option the acoustic absorber is of the slot design type described earlier. The combustion response is described by  $\beta_I$ , and  $\omega$  is the iterated variable. Required input variables are  $\beta_I$ ,  $\beta_R$  (absorber backing distance),  $W_c$  (absorber cavity width),  $L_a$  (absorber aperture length),  $X_A$ ,  $X_B$ ,  $\bar{a}_a$ , (ratio of sound speed in the cavity to sound speed in the main chamber),  $\bar{\rho}_a$  (nondimensional aperture gas density), AMF (aperture mean flow), and  $\varepsilon$  (wave amplitude of the oscillation). Output variables are  $\omega$ ,  $n$  and  $\tau$ ,  $\mu_{qn}$  and  $\beta_L$ , the equivalent absorber admittance for the given geometry.

### Option 4

This option is the same as Option 3 except that  $n$  and  $\tau$  are input and  $\beta_I$  is output.

### Option 5

The last two options use  $\beta_I$  as the iterated variable. They are most useful in generating stability maps in terms of  $n$  and  $\tau$  (Option 5) or real ( $\beta_I$ ) and imag ( $\beta_I$ ), (Option 6). Examples of such stability maps are presented in References 1, 2, 3 and 14. Frequency is used as parameter along these curves.

Option 5 is designed to compute  $\beta_I$  for a given complex frequency and a slot absorber. All the slot absorber parameters necessary for Option 3 must be supplied here as well, in addition to the complex frequency. Output includes  $\mu_{\ell n}$ ,  $\beta_I$ ,  $n$  and  $\tau$ , and  $\beta_L$ , the equivalent liner admittance.

Option 6

This option also uses  $\beta_I$  as the iterated variable. In this option, however, the absorber is characterized by an admittance,  $\beta_L$ , and a length  $W_a = X_B - X_A$ . For this option  $\omega$ ,  $\beta_L$ ,  $X_A$  and  $X_B$  must be supplied and output will give  $\beta_I$ ,  $\mu_{\ell n}$ , and  $n$  and  $\tau$ .

A summary of the principal input and output variables for the six options is given in Table 1 below.

TABLE 1

OPTION	ONE	TWO	THREE	FOUR	OUTPUT
1	real( $\beta_I$ )	imag( $\beta_I$ )	real( $\beta_L$ )	imag( $\beta_L$ )	$\omega$ , $n$ & $\tau$
2	$n$	$\tau$	real( $\beta_L$ )	imag( $\beta_L$ )	$\omega$ , $\beta_I$
3	real( $\beta_I$ )	imag( $\beta_I$ )	BR	AMF	$\omega$ , $n$ & $\tau$ , $\beta_L$
4	$n$	$\tau$	BR	AMF	$\omega$ , $\beta_I$ , $\beta_L$
5	real( $\omega$ )	imag( $\omega$ )	BR	AMF	$\beta_I$ , $n$ & $\tau$ , $\beta_L$
6	real( $\omega$ )	imag( $\omega$ )	real( $\beta_L$ )	imag( $\beta_L$ )	$\beta_I$ , $n$ & $\tau$

For convenience four main input variables are called ONE, TWO, THREE and FOUR both in the program and in the table. These variables represent different quantities in the different options. For example, in Option 1 variable TWO represents the imaginary part of  $\beta_I$ , whereas in Option 2 it represents  $\tau$ , the time lag.

### Choice of Fundamental Acoustic Mode

As was mentioned in the "Theory" section, the success of the iteration process revolves around the assumption that the oscillation with active walls and mean flow is similar to one of the normal acoustic modes (no flow, hard walls) of the combustion chamber. The most useful variable for determining the suitability of an acoustic mode choice is the real part of the complex frequency. As a general rule, when the frequency of oscillation in the combustor is within 10% of a particular acoustic mode frequency, convergence will usually occur if that acoustic mode is used for  $\Omega_{\hat{\ell}\hat{m}\hat{n}}$  in the iterative process. Since the imaginary part of the frequency can be as large as the deviation of the frequency from its acoustic value, nondimensional decay (or growth) rates as large as 0.20 (of the order of  $1000 \text{ sec}^{-1}$  for typical  $\bar{a}^*$  and  $R^*$ ) can occur for these conditions.

For lower acoustic modes there is considerable separation in frequencies. At higher frequencies a given frequency may be close to two (or more) acoustic modes. In this latter case convergence problems can occur and it may be necessary to test all of the possible acoustic modes sequentially. Experience with the program must be the guide in these cases.

For many (if not most) applications the acoustic mode choice is clear. For example, suppose that in a given combustor of diameter 2 ft, length 2 ft and average sound speed,  $\bar{a}^*$ , of 3000 ft/sec, an oscillation of frequency  $5700 \text{ sec}^{-1}$  were observed. The real part of the nondimensional frequency would be  $\omega_R = (5700)/3000 = 1.90$ . This value is within 10% of 1.841, the acoustic frequency of the first transverse mode.  $(J_1(\lambda_{11}r), \ell = 1, m = 1, n = 0, \eta_{\ell mn} = \lambda_{11})$  Hence, when investigating this oscillation using the iterative model,  $\Omega_{\hat{\ell}\hat{m}\hat{n}} = \Omega_{110}$  (i.e.,  $\hat{\ell} = 1, \hat{m} = 1, \hat{n} = 0$ ). Indeed, the

choice of mode would be the same for frequencies between about  $4970 \text{ sec}^{-1}$  and  $6075 \text{ sec}^{-1}$ . On the other hand, if the observed frequency were  $6900 \text{ sec}^{-1}$  the oscillation would be closer in frequency to the combined first transverse, first longitudinal mode frequency of  $7260 \text{ sec}^{-1}$  ( $J_1(\lambda_{11}r)$

$\cos \frac{\pi z}{L}$ ,  $\ell = 1$ ,  $m = 1$ ,  $n = 1$ ,  $\eta_{\ell mn}^2 = \left( \lambda_{11}^2 + \frac{\pi^2}{L^2} \right)$  ) and  $\Omega_{111}$  ( $\hat{\ell} = 1$ ,  $\hat{m} = 1$ ,  $\hat{n} = 1$ ) should be used. When the frequency of oscillation is "in between" two acoustic frequencies and is within 10% of neither, it is usually best to pick the higher mode. For the example given, if  $\omega_R = 6300 \text{ sec}^{-1}$  it would be within 10% of neither the pure first transverse frequency nor the combined first transverse, first longitudinal frequency. The best choice in this case would be  $\Omega_{111}$  rather than  $\Omega_{110}$ .

The choice of acoustic mode is input to the program through the choice of the three integers,  $\hat{\ell}$  (radial),  $\hat{m}$  (azimuthal),  $\hat{n}$  (axial). In the program these are called LHAT, MHAT, and NHAT, respectively.

REFERENCES

1. Mitchell, C. E., W. R. Espander, and M. R. Baer, "Stability of Combustors with Partial Length Acoustic Liners," NASA CR-120889, 1972.
2. Mitchell, C. E., W. R. Espander, and M. R. Baer, "Determination of Decay Coefficients for Combustors with Acoustic Absorbers," NASA CR-120836, 1972.
3. Baer, M. R. and C. E. Mitchell, "A Theoretical Evaluation of Rigid Baffles in the Suppression of Combustion Instability," NASA CR-134986, 1975.
4. Espander, W. R., "Partial Length Liners in Rocket Motors," M.S. Thesis, Dept. of Mechanical Engineering, Colorado State University, 1971.
5. Baer, M. R., "Combustion Instability in Rocket Motors with Distributed Combustion and Acoustic Liners," M.S. Thesis, Dept. of Mechanical Engineering, Colorado State University, 1973.
6. Baer, M. R., "A Theoretical Evaluation of Rigid Baffles in Suppression of Combustion Instability," Ph.D. Thesis, Dept. of Mechanical Engineering, Colorado State University, 1975.
7. Mitchell, C. E., "Stability of Combustors with Partial Length Acoustic Liners," Combustion Science and Technology, Vol. 6, pp. 61-70, 1972.
8. Baer, M. R. and C. E. Mitchell, "Stability of Partially Lined Combustors with Distributed Combustion," AIAA Journal, Vol. 12, No. 4, pp. 475-480, 1974.
9. Mitchell, C. E., "Stability Predictions for Combustors with Acoustic Absorbers and Continuous Combustion Distributions," AIAA Journal, Vol. XIII, No. 8, pp. 1107-1109, 1975.
10. Mitchell, C. E. and Y. Jotiban, "Velocity Dependent Combustion Stability," presented at the 13th JANNAF Combustion Meeting, Monterey, Calif., 1976.
11. Mitchell, C. E., "The Effect of Entropy Waves on High Frequency Pressure Oscillations in Liquid Rocket Motors," Combustion Science and Technology, Vol. 1, pp. 269-274, 1970.
12. Crocco, L. and W. A. Sirignano, Behaviour of Supercritical Nozzles under Three-Dimensional Oscillatory Conditions, AGARDograph No. 117, Butterworths, 1967.
13. Eckert, K. W., "Computer Programs for Linear and Nonlinear Analysis of Rocket Engine Stability," M.S. Thesis, Dept. of Mechanical Engineering, Colorado State University, 1978.

14. Harrje, D. T. ed., F. H. Reardon, assoc. ed., Liquid Propellant Rocket Combustion Instability, NASA SP-194, 1972.
15. Phillips, B., Hannum, N. P., and L. M. Russell, "On the Design of Acoustic Liners for Rocket Engines: Helmholtz Resonators Evaluated with a Rocket Combustor," NASA TN D-5171, April 1969.

Program MODULE

In this section a general description of the program will be given first. Next, discussions of input and output formats will be given. Finally, a sample run will be presented and discussed, and a complete program listing will be given.

General Description of Program

The structure of the programs is as follows. (See Figure 3 for a flow diagram and Table 2 for a listing of program nomenclature.) After the non-default type variables have been declared, and the matrices and arrays have been dimensioned, the values for constants (such as  $PI [\pi]$ ) are stored. Next, values for the two program variables  $K$  and  $IDCR$  are stated. The values for the iterated variable and the percent error in the modulus, and the absolute change in angle, will be printed out the first, last and every  $K^{th}$  iteration.  $IDCR$  is an arbitrary number, after which a percent error in the modulus of over 50% will indicate that the problem is not converging. The values for the two constants  $K$  and  $IDCR$  (ID critical), may need to be changed, but it was felt they would not be changed often enough to warrant including them with the other input data.

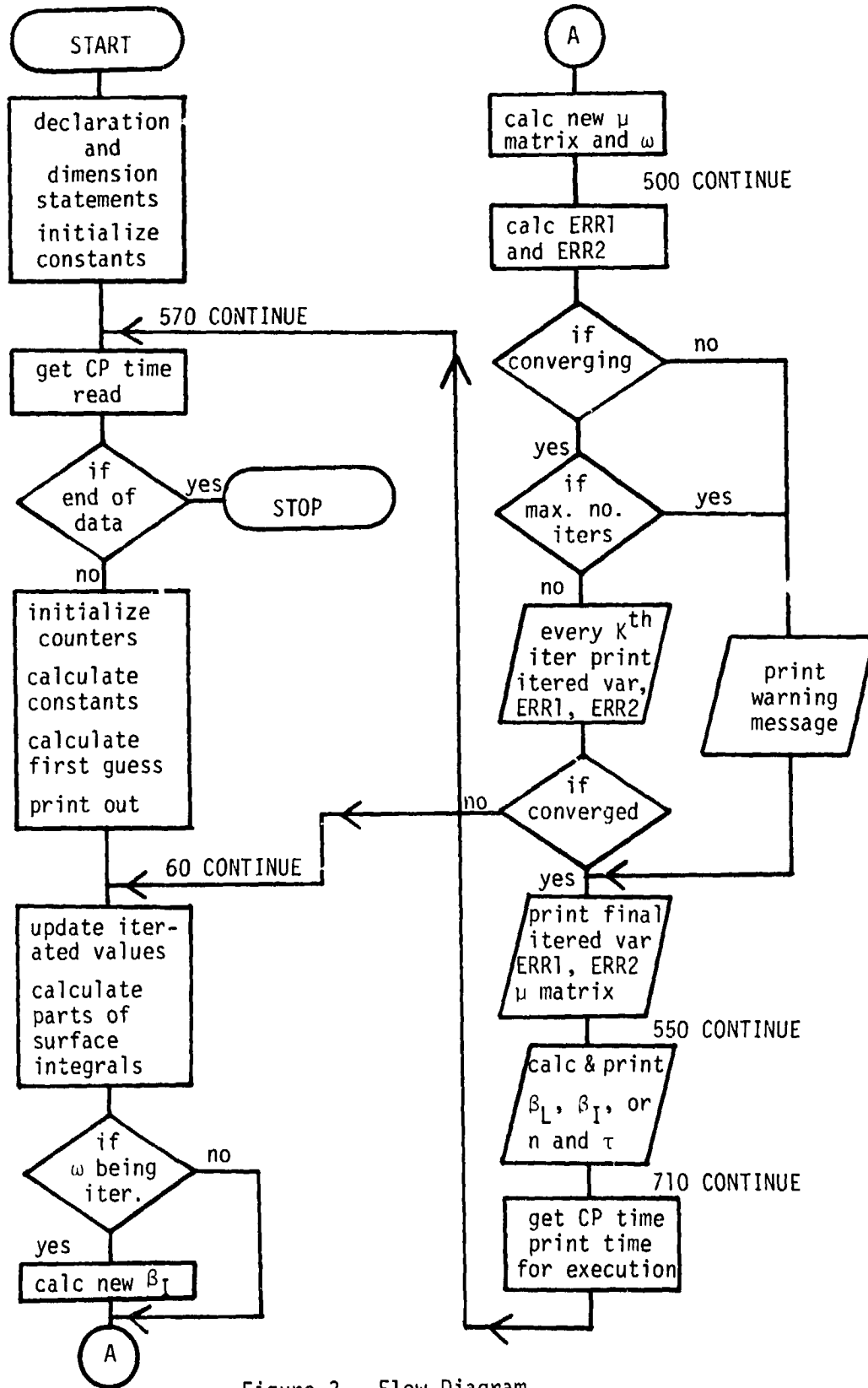


Figure 3. Flow Diagram



TABLE 2

Computer Program Nomenclature

Input Variables

AMF	- aperture mean flow
ACAV	- $A_C$ nondimensional sound speed in slot absorber cavity
BETAI	- $\beta_I$ , acoustic admittance of injector
BETAL	- $\beta_L$ , acoustic admittance of liner
BETAN	- $\beta_N$ , acoustic admittance of nozzle
BR	- backing distance
EPSIL	- $\epsilon$ , amplitude of wave oscillation (only used in calculation of absorber resistance)
ERROR	- acceptable % error in magnitude and absolute change in radians between two successive iterations
FOUR	- OPTION = 1,2 or 6, $\text{imag}(\text{BETAL})$ OPTION = 3,4 or 5, AMF
GAMMA	- $\gamma$ , ratio of specific heats
IDMAX	- maximum number of iterations
IN	- n, interaction index
K	- iterated variable is output every $K^{\text{th}}$ iteration
LENGTH	- L, nondimensional chamber length
LHAT	- $\hat{\ell}$ , radial acoustic mode, integer
LTS	- number of terms in radial direction
MACH	- M, Mach number
MHAT	- $\hat{m}$ , transverse acoustic mode, integer
NHAT	- $\hat{n}$ , longitudinal acoustic mode, integer
NTS	- number of terms in radial direction

- ONE - OPTION = 2 or 3, real(BETAI)  
 OPTION = 2 or 4, IN  
 OPTION = 5 or 6, real(OMEGA)
- OPTION - integer value between 1 and 6, sets which way the problem will be calculated
- ROAP -  $\bar{\rho}_a$ , density in slot absorber aperture
- ROCAV -  $\bar{\rho}_c$ , density in slot absorber cavity
- TAU -  $\tau$ , sensitive time lag
- THREE - OPTION = 1, 2 or 6, real(BETAL)  
 OPTION = 3, 4 or 5, BR
- TWO - OPTION = 1 or 3, imag(BETAI)  
 OPTION = 2 or 4, TAU  
 OPTION = 5 or 6, imag(OMEGA)
- WCAV - nondimensional slot absorber cavity width
- XA - distance from injector to beginning of acoustic liner
- XB - distance from injector to end of acoustic liner.

Program Variables

- A1 -  $\phi$  evaluated at injector
- A2 -  $\phi$  evaluated at nozzle
- BES - 
$$\frac{J_m^2(\lambda_{lm})(\lambda_{lm}^2 - m^2)}{2\lambda_{lm}^2} = \int_0^1 J_m^2(\lambda_{lm} r) r dr$$
- BETA IN - new BETAI
- CIOM -  $i\omega$
- DZP1L -  $\frac{\partial \phi}{\partial z}$  evaluated at liner midpoint
- ETA -  $\eta_{lm}^2$
- ETA1 -  $\eta_{lm}^2$
- ID - iteration counter
- LAMDA2 -  $\lambda_{lm}^2$  where,  $J_m'(\lambda_{lm}) = 0$

LEFF	-	$\ell_{\text{eff}}$
MU	-	$\mu$ matrix
MUX	-	old AMU
NORM	-	$\Lambda_{\ell n}^{1/2}$
NORM1	-	$\Lambda_{\hat{\ell}\hat{n}}^{1/2}$
NPIL2	-	$\left(\frac{\hat{n}\pi}{L}\right)^2$
OMEGAN	-	new OMEGA
PIL	-	$\frac{\pi}{L}$
PIXL	-	$\frac{\pi XL}{L}$
PSI	-	$\psi$
PIL	-	$\phi$ evaluated at liner midpoint
SINJ	-	$\iint dS_{\text{INJ}}$
SLIN	-	$\iint dS_{\text{LIN}}$
SNOZ	-	$\iint dS_{\text{NOZ}}$
VOL	-	$\iiint dV$
WA	-	$W_a$ , width of liner aperture
WVO	-	$\frac{\text{OMEGA}}{\text{ETAT}}$
XL	-	distance from injector to midpoint of liner.

Output Variables (not defined above)

KI	-	Imaginary part of absorber impedance (reactance)
RO	-	$R_0$ real part of absorber impedance (resistance)

A point right after this (570 CONTINUE) is where the program returns to begin execution of a given set of data. The central processor (CP) time is stored at the beginning of execution of a set of data. This time is used to calculate the execution time for the set of data. Next, the data is read in and the counter ID is initialized. The constants for the set of data, such as  $(\frac{n\pi}{L})$ , are then calculated. The first guess for the iterated variable and the  $\mu$  matrix (MU) are calculated, using the separation of variables solution. The input data and first guess are printed out. The setup is now complete, and each iteration returns to a point just below this (60 CONTINUE).

For each iteration the iterated variable is first updated, then variables that are functions of the iterated variable are updated. The  $\mu$  matrix is stored in an extra matrix (MUX). The portion of the program from here to 500 CONTINUE, is designed to evaluate Equations (14) and (15), which give a new  $\mu$  matrix and iterated variable, respectively. The next section, down to 550 CONTINUE, does the following: Calculates the percent error in the modulus (ERR1), and the absolute error in the angle (ERR2). Then checks to see if the problem appears to be converging (ID greater than IDCR and ERR1 greater than 50%), or has gone the maximum number of iterations. If neither of the above has happened, the iterated variable, ERR1 and ERR2 are printed out, if first or  $K^{\text{th}}$  iteration. Then checks to see if the problem has converged (ERR1 and ERR2 are both less than ERROR). If the problem has not converged, the program returns to 60 CONTINUE. If the problem is not converging, has gone the maximum number of iterations, or has converged, the final values for the iterated variable, ERR1, ERR2, and the final  $\mu$  matrix, are printed out. If the problem does not converge, a message is printed out.

The next segment of the program, down to 710 CONTINUE, calculates and prints the other information that is to be output. If a slot absorber is used, an equivalent BETAL is calculated. If IN (n) and TAU ( $\tau$ ) are used, the equivalent BETAI is calculated; otherwise, the corresponding IN and TAU are calculated for the final OMEGA and BETAI. The CP time at the end of the program is stored. Running time is calculated and printed out. The calculations for this set of data are then complete and the program returns to 570 CONTINUE to begin calculations for the next set of data. If no more data is found, the program jumps to 6000 CONTINUE and stops without an error message.

The program is capable of handling up to 10 terms in the radial direction (LTS), 50 terms in the longitudinal direction (NTS), and a transverse mode as high as 4 (MHAT). This should be satisfactory for most cases. However, if the number of terms in the radial or transverse directions must be increased, the Bessel values and Bessel roots for higher modes must be added to subroutines BESVL and BESRT, respectively. Also, all relevant dimension statements must be increased. To increase the number of terms in the longitudinal direction, only the dimensions of MU and MUX must be increased. If MHAT and NHAT are zero (0), LHAT cannot be one (1). This is a trivial case.

The best compromise between good accuracy and fast running time (as discussed previously) occurs with 15-20 terms in the longitudinal direction (NTS), and 3 or 5 terms in the radial direction (LTS). LTS should always be odd, because the series has alternating signs in the radial direction. If the Mach number is greater than .40, more terms should be kept in the longitudinal direction.

Commonly, for liners covering less than one-third of the chamber walls, evaluating the integral over the surface of the liner, by evaluating at the midpoint and multiplying by the width, gives a good approximation to the integral with a big saving in running time. The program is set up to run this way. However, if it is desired to carry the integration out, replace the SLIN card with the CALL LINER card and include SUBROUTINE LINER. (See program listing after 40 CONTINUE.)

The final  $\mu$  matrix, which is printed out, should always be checked to be sure that the term corresponding to the acoustic mode assumed is the largest term in the matrix. If this is not the case, the wrong primary acoustic mode has been assumed, and the answer is not a characteristic of the primary mode assumed.

#### Program Input

The input necessary and the formats for typing the data cards are listed in Table 3. Before the first data card can be typed it must be decided from what is known about the engine (or desired from the calculation) what value OPTION must take. It will be helpful to refer to the "Computational Methods" section in making this determination. Once the value of OPTION is fixed, Table 1 is used to determine which values the variables ONE, TWO, THREE and FOUR must take. The first data card can then be typed.

The second data card contains the model information which must be supplied regardless of option. All inputs are real numbers. The third card contains program variable information, including convergence and matrix size limitation information. All variables on card three are of the integer type. The fourth data card needs to be included only when

TABLE 3

The first card:

<u>COLUMNS</u>	<u>VARIABLE</u>	<u>TYPE</u>
1-20	ONE	Real number
21-40	TWO	Real number
41-60	THREE	Real number
61-80	FOUR	Real number

The second card:

<u>COLUMNS</u>	<u>VARIABLE</u>	<u>TYPE</u>
1-10	real(BETAN)	Real number
11-20	imag(BETAN)	Real number
21-30	GAMMA	Real number
31-40	MACH	Real number
41-50	LENGTH	Real number
51-60	XA	Real number
61-70	XB	Real number
71-80	ERROR	Real number

The third card:

<u>COLUMNS</u>	<u>VARIABLE</u>	<u>TYPE</u>
1-10	LHAT	Integer
11-20	MHAT	Integer
21-30	NHAT	Integer
31-40	LTS	Integer
41-50	NTS	Integer
51-60	IDMAX	Integer
61-70	OPTION	Integer

The fourth card:

<u>COLUMNS</u>	<u>VARIABLE</u>	<u>TYPE</u>
1-10	EPSIL	Real
11-20	ROAP	Real
21-30	ACAV	Real
31-40	ROCAV	Real
41-50	WCAV	Real
51-60	LA	Real

a slot type absorber is present in the combustor. All variables appearing on this card are real.

When setting up the program the correspondence between text and program variables given in Table 2 will be useful. Also, it should be remembered that all variables in the program and text are nondimensional.

### Program Output

The primary outputs of MODULE are the matrix  $\mu_{\ell n}$  and the iterated variable, either  $\omega$  or  $\beta_I$ . Values for these quantities are computed at every step and  $\omega$  (or  $\beta_I$ ) is written out for every  $K^{\text{th}}$  iteration.  $K$  can be changed by the user by replacing a single card. In the program as presented here  $K = 5$ . The first and last iterations of  $\mu_{\ell n}$  are also printed out. The first iteration is a solution with no liner effect, consequently all terms in  $\mu_{\ell n}$  except  $\mu_{\hat{\ell}\hat{n}}$  are null entries.

In addition, all model design variables are printed and labelled according to the program names of Table 2. The example to be discussed next demonstrates the typical form the output takes.

### Sample Run

The combustor used for this sample run has a ratio of length to radius (LENGTH) of 2.0, a mean flow Mach number (MACH) of .3, and a ratio of specific heats (GAMMA) of 1.2. From the Mach number and ratio of specific heats, a nozzle response (BETAN) was calculated using the equation for a short nozzle given in the theory section of this report. The value for BETAN is  $.025 + 0.0i$ . There is an acoustic liner, of known admittance (BETAL), of  $.075 + 0.0i$  in place covering 10% of the cylindrical surface of the chamber, beginning one-tenth of the chamber length downstream of



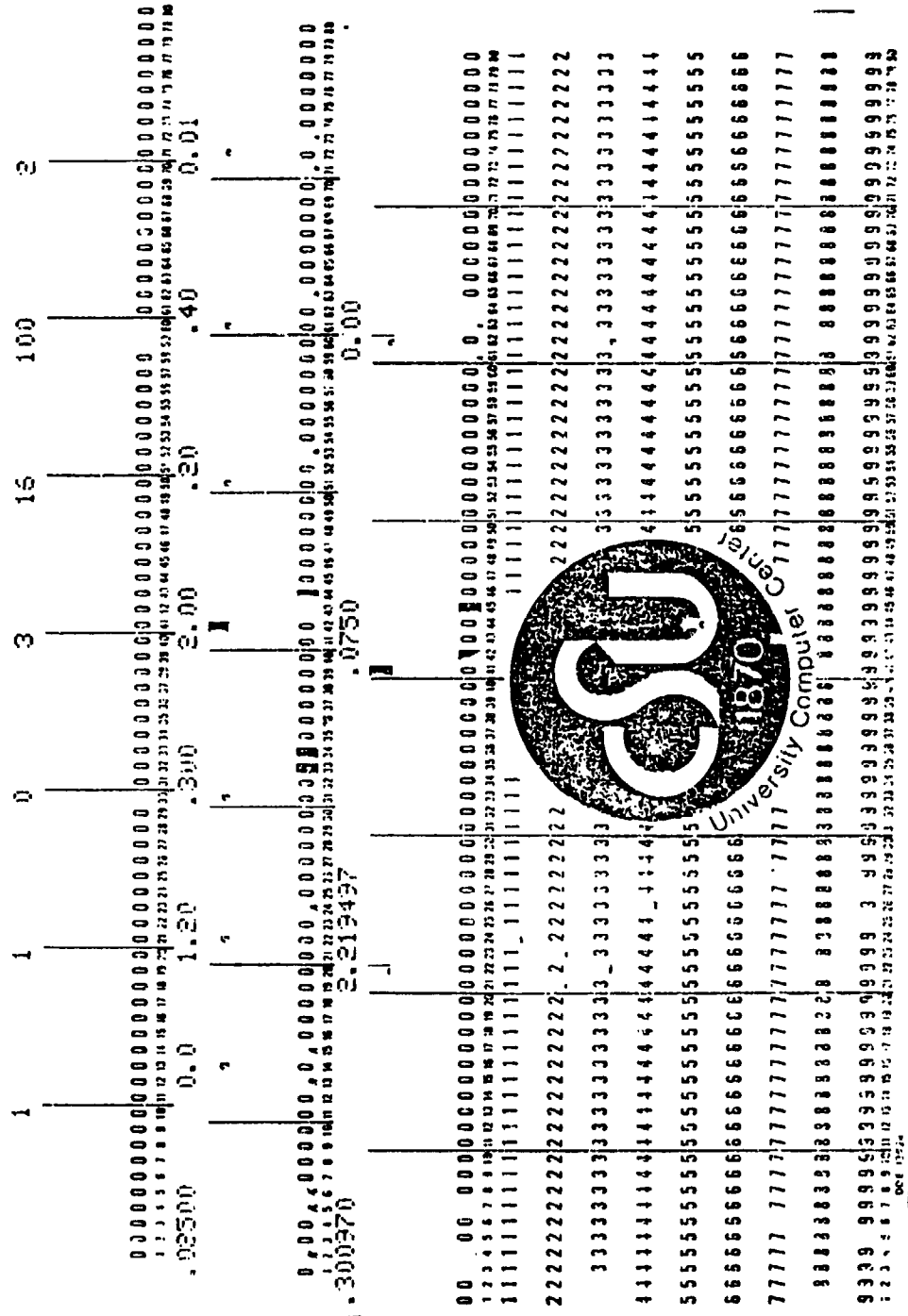
the injector face. Since space dimensions are nondimensionalized by dividing by the chamber radius, and the nondimensional chamber length is 2.0, this gives an  $X_a$  of .2 and an  $X_b$  of .4. The interaction index  $IN(n)$  and sensitive time lag  $TAU(\tau)$  are known to be equal to .30097 and 2.219497, respectively, for the injector response. With the information known about the injector and liner responses and looking at Table 1, it is determined that OPTION must equal 2, and ONE is  $IN$ , TWO is  $TAU$ , THREE is real (BETAL), and FOUR is imag (BETAL). Table 1 shows that  $(\omega)$ , OMEGA, the nondimensional frequency and decay rate, and the effective BETAI for the injector will be calculated. The first transverse acoustic mode is chosen as the primary mode. This corresponds to LHAT = 1, MHAT = 1, and NHAT = 0. A good compromise between running time and accuracy was desired, so by referring to the discussion in this report, the number of terms chosen in the radial direction (LTS) is equal to 3, and the number of terms chosen in the longitudinal direction (NTS) is equal to 16. A high precision is desired, so ERROR is chosen as .01. The maximum number of iterations allowed for this set of data (IDMAX) will be 50. Since an acoustic liner is used, no fourth data card is needed. The input values are then typed up on three data cards, as described in Table 3. The cards as punched and submitted appear in Figure 4.

The output from the sample run is shown in Figure 5. The program first prints out the value of every variable that is input on the data cards. This is to allow for double checking, to be sure all the input data is correct, and also so there is a complete description of the rocket engine that was simulated. Next, the fundamental frequency for the primary mode assumed, and the first guess of the iterated variable, and the  $\mu$

matrix are printed. (These are the separation of variables values.) If OPTIGN is 5 or 6, the first guess of BETAI is printed under the injector response description. Next, the value of the iterated variable is printed for the first, every  $K^{\text{th}}$ , and last iteration. In the sample run  $K=5$ , so the first, fifth and sixth (last) iteration values are printed, along with the errors in the modulus and phase angle of the iterated variable ( $\omega$  in this case). The last  $\mu_{\ell n}$  matrix (iteration 6) is then printed out. It can be seen that the term corresponding to  $\ell = \hat{\ell} = 1$  and  $n = \hat{n} = 0$  is, indeed, the largest. In fact, in this case it is an order of magnitude larger than any other matrix element. The other output information is then printed. In this case, the BETAI calculated from the input IN and TAU and the final OMEGA. The last thing printed out for each set of data is the beginning time TBG, ending time TEND, and execution time TEX, for this set of data.

#### Program Listing

A complete program listing is presented at the end of this report. Comment cards are used liberally and much of the program is self-explanatory. The computer program MODULE conforms to Fortran IV ANSI standards.



ORIGINAL PAGE IS OF POOR QUALITY

Figure 4. Data Cards for Sample Run



ITER	REAL OMEGA	IMAG OMEGA	FPHONS	MODULUS	ANGLE
1	1.79984	.162375E-01		1.007742	.010132
5	1.78849	.177707E-01		.036145	.006235
6	1.78837	.176785E-01		.008533	.000051

THE FINAL MU MATRIX IS AS FOLLOWS

N	L=1	L=2	L=3
0	2.04682	0.	.177943E-03
1	.253107	-.205347	.468422E-03
2	.283958E-01	-.455142E-01	.751661E-03
3	.112628E-01	-.230215E-01	.614310E-03
4	.706775E-02	-.164754E-01	.305828E-03
5	.449316E-02	-.514154E-02	.450557E-04
6	.355532E-02	-.450611E-02	-.117743E-03
7	.252901E-02	-.212447E-02	-.179482E-03
8	.210348E-02	-.331627E-02	-.171944E-03
9	.152422E-02	-.194678E-02	-.125628E-03
10	.129652E-02	-.290075E-02	-.706607E-04
11	.941207E-03	-.203506E-02	-.156449E-04
12	.826920E-03	-.256602E-02	.223523E-04
13	.606747E-03	-.174608E-02	.464229E-04
14	.568850E-03	-.194447E-02	.507790E-04
15	.409482E-03	-.121444E-02	.474611E-04

THE FINAL RETAI IS .096181 .069174

TBG= 14.419 TFM= 15.561 TFX= 1.122

ORIGINAL PAGE IS  
OF POOR QUALITY

Figure 5. (Continued)

PROGRAM MODULE (INPUT,OUTPUT,TAPF5=INPUT,TAPF6=OUTPUT)  
 C\* \*\*\*\*\*  
 C\* COMPUTER PROGRAM MODULE  
 C\* WRITTEN AT COLORADO STATE UNIVERSITY  
 C\* DEPARTMENT OF MECHANICAL ENGINEERING  
 C\* FORT COLLINS, COLORADO 80523  
 C\* SPONSORED BY NASA LEWIS RESEARCH CENTER  
 C\* GRANT NGR 06 - 002 - 095  
 C\* DIRECTED BY RICHARD J. PRIEM  
 C\* THIS PROGRAM IS DOCUMENTED IN THE MASTERS THESIS OF KURTIS W. ECKFRT  
 C\* 1976, COLORADO STATE UNIVERSITY, AND CONFORMS TO ALL FORTRAN IV ANSI  
 C\* STANDARDS.

C\* IT IS WRITTEN TO GIVE A LINEAR ANALYSIS OF HIGH FREQUENCY  
 C\* COMBUSTION STABILITY IN LIQUID PROPELLANT ROCKET ENGINES. THE  
 C\* PHYSICAL MODEL USED IS A RIGHT CIRCULAR CYLINDER. THE COMBUSTION  
 C\* IS MODELED AS EITHER AN ARBITRARY ACOUSTIC ADMITTANCE (BETA1), OR  
 C\* BY THE CROCCO SENSITIVE TIME LAG THEORY (IN AND TAU). THE NOZZLE  
 C\* IS MODELED AS AN ARBITRARY ACOUSTIC ADMITTANCE (BETA2). THE LINER  
 C\* IS MODELED AS EITHER AN ARBITRARY ACOUSTIC ADMITTANCE (BETA) OVER  
 C\* SOME PORTION OF THE CYLINDRICAL WALL OF THE CHAMBER, OR AS A SLOT  
 C\* ABSORBER.

C\* THIS PROGRAM HAS BEEN WRITTEN TO SOLVE FOR THE NONDIMENSIONAL  
 C\* COMPLEX FREQUENCY IF ALL THE BOUNDARY RESPONSES ARE GIVEN, OR SOLVE  
 C\* FOR THE INJECTOR RESPONSE IF THE COMPLEX FREQUENCY AND LINER AND  
 C\* NOZZLE RESPONSES ARE GIVEN. THIS LEADS TO A TOTAL OF 6 OPTIONS  
 C\* FOR RUNNING THE COMPUTER PROGRAM.

C\* FOLLOWING IS A TABLE TO USE IN DETERMINING WHICH VALUE OF OP-  
 C\* TION TO USE. THE TABLE SHOWS WHAT INFORMATION MUST BE GIVEN AND  
 C\* WHAT WILL BE CALCULATED FOR EACH OPTION. R() MEANS THE REAL PART OF  
 C\* THE COMPLEX VALUE INSIDE THE PARENTHESES, AND I() MEANS THE IMAGIN-  
 C\* ARY PART OF THE COMPLEX VALUE INSIDE THE PARENTHESES.

C\* TABLE OF OPTIONS

OPTION	ONE	INPUT VARIABLES			FOUR	OUTPUT VARIABLES
		TWO	THREE			
1	R(BETA1)	I(BETA1)	R(BETA2)	I(BETA2)	OMEGA, IN & TAU	
2	IN	TAU	R(BETA2)	I(BETA2)	OMEGA, BETA1	
3	R(BETA1)	I(BETA1)	RR	AMF	OMEGA, BETA1, IN & TAU	
4	IN	TAU	RR	AMF	OMEGA, BETA1, BETA2	
5	R(OMEGA)	I(OMEGA)	RR	AMF	BETA1, BETA2, IN & TAU	
6	R(OMEGA)	I(OMEGA)	R(BETA2)	I(BETA2)	BETA1, IN & TAU	

C\* FOLLOWING IS A TABLE LISTING THE INPUT VARIABLES THAT ARE PUT  
 C\* ON EACH DATA CARD. AFTER THE VARIABLE NAME THE TYPE OF VARIABLE IS  
 C\* SHOWN. THEN A BRIEF DESCRIPTION OF THE VARIABLE IS GIVEN. THEN AT  
 C\* THE END OF THE LINE ARE THE COLUMNS OF THE DATA CARD WHICH THE VALUE  
 C\* FOR THIS VARIABLE MUST BE TYPED IN. ALWAYS BE SURE TO RIGHT JUSTIFY  
 C\* INTEGER VALUES. THE FOURTH DATA CARD IS USED ONLY WHEN OPTION EQUALS  
 C\* 3, 4 OR 5.

C\* LIST OF INPUT VARIABLES

FIRST CARD			
ONE	REAL	SEE TABLE ABOVE	1-20
TWO	REAL	SEE TABLE ABOVE	21-40
THREE	REAL	SEE TABLE ABOVE	41-60
FOUR	REAL	SEE TABLE ABOVE	61-80
SECOND CARD			
BETA2	COMPLEX	ACOUSTIC ADMITTANCE OF NOZZLE MODULUS LESS THAN .5	1-10 & 11-20
GAMMA	REAL	IF NO KNOWN VALUE USE SHORT NOZZLE RATIO OF SPECIFIC HEATS	21-30

C*	MACH	REAL	MEAN FLOW MACH NUMBER 0 < MACH < .5	31-40
C*	LENGTH	REAL	LENGTH OF CHAMBER/RADIUS OF CHAMBER	41-50
C*	XA	REAL	DISTANCE FROM INJECTOR FACE TO START OF LINER/RADIUS OF CHAMBER	51-60
C*			0 < XA < XB	
C*	XH	REAL	DISTANCE FROM INJECTOR FACE TO END OF LINER/RADIUS OF CHAMBER	61-70
C*			XA < XB < LENGTH	
C*	ERROR	REAL	MAXIMUM ALLOWABLE % ERROR IN MODULUS OR ABSOLUTE DIFFERENCE IN RADIANS OF ANGLE TO DETERMINE CONVERGENCE OF ITERATED VARIABLE 10E-5 < ERROR < 1	71-80
C*	THIRD CARD			
C*	LHAT	INTEGER	ASSUMED MODE IN RADIAL DIRECTION TYPICALLY 1 OR 2	1-10
C*	MHAT	INTEGER	ASSUMED MODE IN TRANSVERSE DIRECTION TYPICALLY 0, 1 OR 2	11-30
C*	NHAT	INTEGER	ASSUMED MODE IN LONGITUDINAL DIRECTION TYPICALLY 0, 1 OR 2	21-30
C*	LTS	INTEGER	NUMBER OF TERMS KEPT IN RADIAL DIRECTION ODD < 10 TYPICALLY 3 OR 5	31-40
C*	NTS	INTEGER	NUMBER OF TERMS KEPT IN LONG. DIRECTION NTS < 50 TYPICALLY 15 < NTS < 20	41-50
C*	IDMAX	INTEGER	MAXIMUM NUMBER OF ITERATIONS ALLOWED TYPICALLY 100	51-60
C*	OPTION	INTEGER	SEE TABLE ABOVE 1, 2, 3, 4, 5, OR 6	61-70
C*	FOURTH CARD			
C*	EPSIL	REAL	WAVE AMPLITUDE	1-10
C*	ROAP	REAL	APERTURE DENSITY RATIO	11-20
C*	ACAV	REAL	CAVITY SOUND SPEED	21-30
C*	ROCAV	REAL	CAVITY DENSITY RATIO	31-40
C*	WCAV	REAL	CAVITY WIDTH	41-50
C*	LA	REAL	APERTURE LENGTH	51-60

```

C*****
C
C   REAL IN1,INO,IN,NPIL2,LENGTH,LFF,LA,LAMDA,LAMDA2,NORM,MACH,MACH2
C   1,NORM1
C   INTEGER OPTION,CHECK
C
C   COMPLEX HETAN,BETAL,BETAI,MU,MUX,OMEGA,OMEGAN,CIOM,CERROR,CI
C   1,CZERO,F1,HLINE,P1L,DZPIL,PRES,A1,A2,SUM1,SUM2,
C   2AV,BASFW,BETAIN,WVO
C   COMPLX CTERM,B1,B2,H1SQ,H2SQ,A,EXP1,EXP2,TERM1,TERM2,VOL,SINJ,
C   1SNOZ,SINJR,SLIN,PSI
C
C   DIMENSION MU(50,10),MUX(50,10),A1(10),A2(10)
C
C   INITIALIZE CONSTANTS
C
C   PI=3.14159265359
C   CI=CMPLX(0.,1.)
C   CZERO = CMPLX(0.,0.)
C   IDCR= 20
C   K=5
C
C   READ IN DATA
C
C 570 CONTINUE
C   CALL SECOND (TRGN)
C   READ (5,100) ONE,TWO,THREE,FOUR,BETAN,GAMMA,MACH,LENGTH,XA,XH,
C   1ERROR,LHAT,MHAT,NHAT,LTS,NTS,IDMAX,OPTION
C   IF (EOF(5)) 6000,580,6000
C 580 CONTINUE
C   IF (OPTION.GE.3.AND.OPTION.LE.5) READ (5,101) EPSIL ,ROAP,
C   1 ACAY,ROCAV,WCAV,LA
C
C   INITIALIZE VARIABLES

```

```
ANG1= 0.0
ID=0
IF (OPTION.EQ.1.OR.OPTION.EQ.3) BETAIN= CMPLX(ONE,TWO)
IN= ONE
TAU= TWO
BETAL= CMPLX(THREE,FOUR)
BR= THREE
AMF= FOUR

C
C
C
CALCULATE CONSTANTS
PIL= PI/LENGTH
NPIL2= (FLOAT(NHAT)*PIL)**2
WA = XB -XA
XL = XA + WA/2.0
PIXL=PI*XL/LENGTH
LAMDA2=BESRT(MHAT,LHAT)**2
IF (OPTION.LE.2.OR.OPTION.EQ.6) GO TO 30
LEFF = LA + 0.375*0.85*WA*( 1. - 0.7 *SQRT( WA/WCAV))
30 CONTINUE
ETA1= SQRT(LAMDA2 + NPIL2)

C
C
C
CALCULATE FIRST GUFSS FOR ITERATED VARIABLES
OMEGAN= .9R*CMPLX(ETA1,0.0)
IF (OPTION.GE.5) OMEGAN= CMPLX(ONE,TWO)
WWO=OMEGAN/ETA1
IF (OPTION.EQ.2.OR.OPTION.EQ.4) BETAIN=MACH*(1./GAMMA - IN*(1. -C
1EXP(-CI*OMEGAN*TAU)))

C
C
C
CHECK VALUES OF INPUT VARIABLES
TERM= WA*REAL(BETAL)
CHECK= 0
IF (MACH.LE.0.0.OR.MACH.GE.1.0) CHECK= 1
IF (MACH.GT.0.5) WRITE (6,900) MACH
IF (GAMMA.LT.1.0.OR.GAMMA.GT.1.57) CHECK= 1
IF (GAMMA.LT.1.1) WRITE (6,920) GAMMA
IF (LENGTH.LE.0.0) CHECK= 1
IF (LENGTH.GE.3.0) WRITE (6,902) LENGTH
IF (LENGTH.LE.1.0) WRITE (6,922) LENGTH
IF (REAL(BETAN).LT.0.0) CHECK= 1
IF (REAL(BETAN).GT.0.3) WRITE (6,906) BETAN
IF (OPTION.GT.2.AND.OPTION.LT.6) GO TO 44
IF (TERM.LT.0.0) CHECK= 1
IF (TERM.GT.0.3) WRITE (6,904) TERM
44 CONTINUE
IF (OPTION.NE.2.AND.OPTION.NE.4) GO TO 46
IF (IN.LT.0.0.OR.TAU.LT.0.0) CHECK= 1
IF (IN.GT.3.0) WRITE (6,912) IN
IF (TAU.GT.4.0) WRITE (6,914) TAU
46 CONTINUE
IF (OPTION.NE.1.AND.OPTION.NE.3) GO TO 47
IF (CAHS(BETAIN).GT.2.0) WRITE (6,916) BETAIN
47 CONTINUE
IF (XA.LT.0.0.OR.XA.GT.XR.OR.XR.GT.LENGTH) CHECK= 1
IF (WA.GT.0.5) WRITE (6,930) WA
IF (LHAT.LE.7.AND.MHAT.LE.4.AND.NHAT.LE.10) GO TO 48
CHECK= 1
WRITE (6,980) LHAT,MHAT,NHAT
48 CONTINUE
IF (NIS.LI.NHAT + 10) WRITE (6,940) NIS,NHAT
IF (LTS.LI.LHAT + 2) WRITE (6,942) LTS,LHAT
IF (REAL(WWO).GT.1.1) WRITE (6,950)
IF (OPTION.LT.3.OR.OPTION.EQ.6) GO TO 49
IF (EPSIL.LT.0.0.OR.RR.LE.0.0.OR.AMF.LT.0.0) CHECK= 1
IF (RR.GT.0.2) WRITE (6,908) RR
IF (AMF.GT.0.2) WRITE (6,910) AMF
49 CONTINUE
IF (CHECK.EQ.1) WRITE (6,960) MACH,GAMMA,LENGTH,BETAL,BETAN,XA,XR,
1IN,TAU
IF (CHECK.EQ.1) GO TO 710
```



C  
C  
C

INITIALIZE FIRST GUESS OF MU MATIX

```

DO 50 N1=1,NTS
DO 50 L=1,LTS
MU(N1,L)= CZERO
50 CONTINUE.

```

C  
C  
C

THE TILDA SOLUTION

```

RV= BESVL(MHAT,LHAT)
EP= 1.
IF (NHAT.EQ.0) EP= 2.
RES1= RV*BV*(LAMDA2 = FLOAT(MHAT)**2)/2./LAMDA2
NORM1= SQRT(RES1*EP*LENGTH/2.)
MACH2= MACH*MACH
DO 75 N= 1,100
CTERM= MACH2*OMEGAN*OMEGAN+(MACH2-1.)*(LAMDA2-OMEGAN*OMEGAN)
CTERM= CSQRT(CTERM)
R1= (MACH*OMEGAN+CTERM)/(1.-MACH2)
R2= (MACH*OMEGAN-CTERM)/(1.-MACH2)
R1SQ=R1*B1
R2SQ=R2*B2
A=- (CEXP(CI*LENGTH*(R1-R2))*(R1+BETAN*GAMMA*(OMEGAN+MACH*R1)) /
1 (R2+BETAN*GAMMA*(OMEGAN+MACH*R2)))
EXP1=CEXP(CI*LENGTH*R1)
EXP2=CEXP(CI*LENGTH*R2)
TERM1= B1*(EXP1*(-1.)**NHAT-1.)/(R1SQ-NPIL2)
TERM2= A*B2*(EXP2*(-1.)**NHAT-1.)/(R2SQ-NPIL2)
VOL= MACH*(MACH*(R1SQ*TERM1+R2SQ*TERM2)+2.*OMEGAN*(R1*TERM1+
1 R2*TERM2))
SINJ=GAMMA*(OMEGAN*(1.+A)+MACH*(R1+A*R2))
SINJB=BETAIN*SINJ
SNOZ= BETAN*GAMMA*(-1.)**NHAT*(OMEGAN*(EXP1+A*EXP2)+MACH*(R1*
1 EXP1 + A*B2*EXP2))
IF (OPTION.GE.5) GO TO 70
OMEGA= -(VOL + SINJB + SNOZ)/(TERM1 + TERM2) + ETA1**2
OMEGA= CSQRT(OMEGA)
IF (OPTION.EQ.2. OR.OPTION.FO.4) BETAIN= MACH*(1./GAMMA-IN*(1.-C
1 EXP(-CI*OMEGAN*TAU)))
CERROR= OMEGA - OMEGAN
OMEGAN= OMEGA
IF ( ABS(REAL(CERROR)).GT.0.0001) GO TO 75
IF ( ABS(AIMAG(CERROR)).GT.0.0001) GO TO 75
GO TO 71
70 CONTINUE
BETAIN= ((ETA1**2- OMEGAN**2)*(TERM1 + TERM2) - VOL - SNOZ)/SINJ
GO TO 71
75 CONTINUE
71 CONTINUE
PSI= (-2.*CI/LENGTH)*(TERM1 + TERM2)
IF (NHAT.EQ.0) PSI= PSI/2.
MU(NHAT+1,LHAT)= CMPLX(1./NORM1,0.0)
DO 55 N1=1,NTS
N= N1-1
RN= FLOAT(N)
IF (N.EQ.NHAT) GO TO 55
TERM= FLOAT((-1)**N)
NPIL2= (RN*PIL)**2
FP= 1.
IF (N.FO.0) EP= 2.
NORM= RES1*EP*LENGTH/2.
ETA= LAMDA2 + NPIL2
TERM1= (EXP1*TERM - 1.)/(R1SQ - NPIL2)
TERM2= (EXP2*TERM - 1.)/(R2SQ - NPIL2)
VOL= MACH2*(R1SQ*B1*TERM1 + A*B2SQ*B2*TER .2) + 2.*MACH*OMEGAN*(R
1 R1SQ*TERM1 + A*B2SQ*TERM2)
SNOZ= BETAN*GAMMA*TERM*(OMEGAN*(EXP1 +A*EXP2) + MACH*(R1*EXP1+
1 A*B2*EXP2))
SINJB= BETAIN*GAMMA*(OMEGAN*(1. +A) + MACH*(R1 + A*R2))
MU(N1,LHAT)= BES1*CI*(VOL+ SNOZ +SINJB)/(OMEGAN**2 -ETA)/NORM1
1/NORM/PSI

```

```
55 CONTINUE
C
C   PRINT OUT PROBLEM DESCRIPTION AND INITIAL VALUFS
C
WRITE (6,1)
WRITE (6,200)
WRITE (6,210) LENGTH,MACH,LHAT,MHAT,NHAT,OPTION,GAMMA
WRITE (6,220)
WRITE (6,230) XA,XR,WA
IF (OPTION.GE.3.AND.OPTION.LE.5) GO TO 20
WRITE (6,235) BETAL
GO TO 22
20 CONTINUE
WRITE (6,240) BR,AMF,LA,LEFF,EPSIL,WCAV,ROCAV,ROAP,ACAV
22 CONTINUE
WRITE (6,250)
IF (OPTION.GE.5) GO TO 10
IF (OPTION.EQ.2.OR.OPTION.EQ.4) GO TO 12
WRITE (6,260) HETAIN
GO TO 14
12 CONTINUE
WRITE (6,270) IN,TAU,HETAIN
GO TO 14
10 CONTINUE
WRITE (6,280) OMEGAN,WWO
WRITE (6,285) HETAIN
14 CONTINUE
WRITE (6,290)
WRITE (6,310) BETAN
WRITE (6,320) LTS,NTS,IDMAX,ERROR
WRITE (6,300) ETAL
WRITE (6,305) OMEGAN
WRITE (6,105)
DO 35 N=1,NTS
NM1=N-1
WRITE (6,104) NM1,(MU(N,L),L=1,LTS)
35 CONTINUE
WRITE (6,2)
IF (OPTION.LE.4) WRITE (6,350)
IF (OPTION.GE.5) WRITE (6,360)
C
C   BEGIN ITERATION
C
60 CONTINUE
C
C   UPDATE ITERATED VARIABLES
C
IF (OPTION.EQ.2.OR.OPTION.EQ.4) HETAIN=MACH*(1./GAMMA - IN*(1.-C
1EXP(-CI*OMEGAN*TAU)))
HETAI=HETAIN
OMEGA=OMEGAN
CIOM=CI*OMEGA
WR=REAL(OMEGA)
C
C   STORE NEW MU MATRIX IN EXTRA MATRIX
C
DO 800 N1=1,NTS
DO 800 L=1,LTS
MUX(N1,L)=MU(N1,L)
800 CONTINUE
C
C   CALCULATE PHI AT INJECTOR, NOZZLE,MIDPOINT OF LINER
C
PIL=CZERO
DZPIL=CZERO
DO 130 L=1,LTS
SUM1=CZERO
SUM2=CZERO
BV=BESVL(MHAT,L)
```

```
DO 120 N1=1,NTS
N= N1 - 1
RN= FLOAT(N)
AV=MU(N1,L)
SUM1= SUM1+AV
SUM2= SUM2 + AV*FLOAT((-1)**N)
PIL=PIL+BV+AV*COS(RN*PIXL)
DZPIL= DZPIL + HV*AV*RN*PIL*SIN(RN*PIXL)
120 CONTINUE
C
A1(L)=SUM1
A2(L)=SUM2
130 CONTINUE
C
C
CALCULATE BLINER FOR MIDPOINT OF LINER
C
PRES= GAMMA*(CIOM*PIL - MACH*DZPIL)
RLINER = WA*PRES*BETAL
C
C
CALCULATE BLINER FROM LINER GEOMETRY
C
IF (OPTION.LE.2.OR.OPTION.FQ.6) GO TO 3000
AK1 = LEFF*WR*ROAP-WA*ACAV*POCAV/WR/WCAV/RR *ACAV
PRE1 = CABS(PRES)
RO=1.
RO1 = RO
C
DO 133 I=1,100
F = SQRT( 1.0 + (AK1/RO)**2 )
BASE= EPSIL*PRE1/F/GAMMA
RO = SQRT( 0.8*( 1.5*AMF*RO + BASE ) )
RO2 = ABS( (RO1-RO)/RO1 )
IF(RO2.LT.1.0E-04) GO TO 134
RO1= RO
133 CONTINUE
C
134 CONTINUE
RLINER = WA*PRES/GAMMA/(RO+CI*AK1)
3000 CONTINUE
IF (OPTION.LE.4) GO TO 3001
C
C
CALCULATE NEW BETAT
C
VOL= CZERO
DO 45 N1=2,NTS
NM1= N1-1
IF (NM1.EQ.NHAT) GO TO 45
K1= NM1 + NHAT
K2= NM1 - NHAT
L1=(-1)**K1-1
L2=(-1)**K2-1
LSUM=L1+L2
IF(LSUM.EQ.0) GO TO 45
C1= FLOAT(L1)/FLOAT(K1)
C2= FLOAT(L2)/FLOAT(K2)
SC= (C1 + C2)*FLOAT(NM1)
VOL= VOL + MUX(N1,NHAT)*SC
45 CONTINUE
VOL= VOL*MACH*CIOM*BFS1
VOL= VOL - MUX(NHAT+1,LHAT)*BFS1*(MACH*FLOAT(NHAT)*PIL)**2*LENGTH
1/2.
VOL= VOL/NORM1
SNOZ= BETAN*GAMMA*CIOM*BES1*A2(LHAT)*FLOAT((-1)**NHAT)/NORM1
SLIN= BLINER*BESVL(MHAT,LHAT)*COS(FLOAT(NHAT)*PIXL)/NORM1
SINJ= GAMMA*CIOM*BFS1*A1(LHAT)/NORM1
BETAIN= (OMEGA**2 -FTA1**2 -VOL -SNOZ -SLIN)/SINJ
3001 CONTINUE
C
C
START DU LOOP FOR L SUMMATION
C
DO 500 L=1,LTS
```



```
C
C
C FOR PRIMARY MODE CALCULATE MU TERM AND NEW OMEGA
MU(NX,L)= CMPLX(1./NORM,0.0)
IF (OPTION.LE.4) OMEGAN= CSQRT(F) + ETA)
500 CONTINUE
ID= ID + 1
IK= IFIX(FLOAT(ID)/FLOAT(K))
RK= FLOAT(ID)/FLOAT(K)
RK= RK - FLOAT(IK)
IF (OPTION.LE.4) GO TO 4000
C
C
C CHECK FOR CONVERGENCE ON BETAI
CERROR= BETAIN - BETAI
ERR1= CABS(CERROR)/CABS(BETAIN)*100.
ANG2= ATAN(AIMAG(BETAIN)/REAL(BETAIN))
ERR2= ABS(ANG2 - ANG1)
ANG1= ANG2
IF (ID.GE.IDCR.AND.ERR1.GE.50.) GO TO 540
IF (ID.GE.IDMAX) GO TO 540
IF (ID.EQ.1.OR.RK.LE.0.0001) WRITE (6,666) ID,BETAIN,ERR1,ERR2
IF (ID.LT.5.OR.ERR1.GT.ERROR.OR.ERR2.GT.ERROR) GO TO 60
GO TO 545
540 CONTINUE
WRITE (6,365)
545 CONTINUE
WRITE (6,666) ID,BETAIN,ERR1,ERR2
GO TO 550
C
C
C CHECK FOR CONVERGENCE ON OMEGA
4000 CONTINUE
CERROR = OMEGAN - OMEGA
ANG2= PI/2.
ERR1= 0.0
IF (CABS(OMEGAN).EQ.0.0) GO TO 43
ERR1= CABS(CERROR)/CABS(OMEGAN)*100.
IF (REAL(OMEGAN).EQ.0.0) GO TO 43
ANG2= ATAN(AIMAG(OMEGAN)/REAL(OMEGAN))
43 CONTINUE
ERR2= ABS(ANG2-ANG1)
ANG1= ANG2
IF (ID.GE.IDCR.AND.ERR1.GE.50.) GO TO 547
IF (ID.GE.IDMAX) GO TO 547
IF (ID.EQ.1.OR.RK.LE.0.0001) WRITE (6,666) ID,OMEGAN,ERR1,ERR2
IF (ID.LT.5.OR.ERR1.GT.ERROR.OR.ERR2.GT.ERROR) GO TO 60
GO TO 546
547 CONTINUE
WRITE (6,365)
546 CONTINUE
WRITE (6,666) ID,OMEGAN,ERR1,ERR2
550 CONTINUE
IF (ID.GE.IDMAX) WRITE(6,390)
C
C
C PRINT OUT FINAL MU MATRIX
WRITE (6,2)
WRITE(6,110)
DO 106 N1=1,NTS
N= N1 - 1
WRITE (6,104) N,(MU(N1,L),L=1,LTS)
106 CONTINUE
WRITE(6,2)
C
C
C CALCULATE AND PRINT EQUIVALENT BETAL IF CALCULATED FROM GEOMETRY
IF (OPTION.GE.3.AND.OPTION.LE.5) WRITE (6,501) RO,AK1
IF (OPTION.GE.3.AND.OPTION.LE.5) BETAL= BLINER/WA/PRES
IF (OPTION.GE.3.AND.OPTION.LE.5) WRITE (6,370) BETAL
```

C  
C  
C

CALCULATE AND PRINT EITHER N, TAU OR BETAI WHICHEVER IS APPROPRIATE

```

IF (OPTION.NE.1.AND.OPTION.NE.3) WRITE (6,380) BETAIN
IF (OPTION.EQ.2.OR.OPTION.FQ.4) GO TO 710
WR= REAL (OMEGAN)
WI= AIMAG(OMEGAN)
BIR= REAL(BETAIN)
BII= AIMAG(BETAIN)
INO= -((BIR - MACH/GAMMA)**2+BII**2)/(2.*MACH*(BIR-MACH/GAMMA))
DO 11 N=1,3
RN= FLOAT(N) - 1.
TAU0= ABS(ASIN(-BII/MACH/INO) + RN*PI)/WR
BETAIN= MACH*(1./GAMMA - INO*(1. - CEXP(-CI*OMEGAN*TAU0)))
ERR1= ABS((BIR - REAL(BETAIN))/BIR)
ERR2= ABS((BII - AIMAG(BETAIN))/BII)
IF (ERR1.LE.0.30.AND.ERR2.LE.0.30) GO TO 19
RN= -RN
TAU0= ABS(ASIN(-BII/MACH/INO) + RN*PI)/WR
BETAIN= MACH*(1./GAMMA - INO*(1. - CEXP(-CI*OMEGAN*TAU0)))
ERR1= ABS((BIR - REAL(BETAIN))/BIR)
ERR2= ABS((BII - AIMAG(BETAIN))/BII)
IF (ERR1.LE.0.30.AND.ERR2.LE.0.30) GO TO 19
11 CONTINUE
WRITE (6,3)
GO TO 710
19 CONTINUE
WRITE (6,386) RN,INO,TAU0,BETAIN
IF (WI.EQ.0.0) GO TO 710
RN= -3.0
18 CONTINUE
RN= RN + 1.0
IF (RN.GE.3.5) GO TO 17
IN1= INO
TAU1= TAU0
DO 15 I=1,200
TERM= BIR - MACH/GAMMA + MACH*IN1
TAU= ABS(ATAN(-BII/TERM) + RN*PI)/WR
IN= TERM/(MACH*EXP(WI*TAU)*COS(WR*TAU))
IF (ABS(TAU - TAU1).LT.1.E-5.AND.ABS(IN - IN1).LT.1.E-5.AND.I.GE.5)
1 GO TO 16
IF (I.GE.5.AND.IN.LT.0.0) GO TO 16
IF (I.GE.5.AND.TAU.LT.0.0) GO TO 16
IN1= IN
TAU1= TAU
15 CONTINUE
16 CONTINUE
IF (IN.LT.0.00.OR.TAU.LT.0.00) GO TO 18
BETAIN= MACH*(1./GAMMA - IN*(1. - CEXP(-CI*OMEGAN*TAU)))
ERR1= ABS((BIR - REAL(BETAIN))/BIR)
ERR2= ABS((BII - AIMAG(BETAIN))/BII)
IF (ERR1.GE.0.15.OR.ERR2.GE.0.15) GO TO 18
WRITE (6,385) RN,IN,TAU,BETAIN
GO TO 710
17 CONTINUE
SWT= SIN(WR*TAU0)
CWT= COS(WR*TAU0)
EWT= EXP(WI*TAU0)
EC1= EWT*CWT - 1.
WTW= WR/TAN(WR*TAU0) + WI
TAU1= (BIR/MACH/EC1 + BII/MACH/EWT/SWT - 1./GAMMA/EC1)/(INO*
1 (EWT*(WI*CWT - WR*SWT)/EC1 - WTW))
IN1= -BII/MACH/EWT/SWT - INO - INO*TAU1*WTW
TAU= TAU0 + TAU1
IN= INO + IN1
BETAIN= MACH*(1./GAMMA - IN*(1. - CEXP(-CI*OMEGAN*TAU)))
WRITE (6,5) IN,TAU,BETAIN

```

C  
C  
C

CALCULATE RUNNING TIME FOR THIS SET OF DATA

```

710 CONTINUE
CALL SECOND (TEND)

```

```

      TEX= TEND - TBGN
      WRITE (6,80) TBGN,TEND,TEX
C C C
      RETURN FOR NEXT SET OF DATA
      GO TO 570
6000 CONTINUE
      STOP
C C C
      FORMAT STATEMENTS
1  FORMAT (1H1,///)
2  FORMAT(///)
3  FORMAT (48H NO AND TAUO WOULD NOT CONVERGE FOR THIS PROBLEM,/)
4  FORMAT (66H N AND TAU DID NOT CONVERGE SO A LINEARIZATION SOLUTI
10N IS GIVEN,/)
5  FORMAT (61H N AND TAU DID NOT CONVERGE SO A LINEARIZED SOLUTION I
1 GIVEN,/,5X,29HTHE CALCULATED N AND TAU ARE ,F9.6,5H AND ,F9.6,/.
25X,18HGIVING A BETAI OF ,F9.6,3X,F9.6,///)
80  FORMAT (7H TBG=,F8.3,6H TEND=,F8.3,5H TEX=,F8.3,///)
100 FORMAT (4F20.10,/,8F10.8,/,7I10)
101 FORMAT(8F10.8)
104 FORMAT(2X,13,3X,8G13.6,/, (8X,8G13.6,/)
105 FORMAT (49H THE FIRST GUESS FOR THE MU MATRIX IS AS FOLLOWS,//5X,
11HN,16X,3HL=1,24X,3HL=2,24X,3HL=3,/)
110 FORMAT (35H THE FINAL MU MATRIX IS AS FOLLOWS,//5X,1HN,16X,3HL=1,
124X,3HL=2,24X,3HL=3,/)
200 FORMAT(85H THE FOLLOWING CALCULATIONS ARE MADE FOR A COMRUSTOR WIT
1H THE FOLLOWING CONFIGURATION,///)
210 FORMAT(20X,23HTHE LENGTH TO RADIUS = ,F9.6,24X,20HMEAN FLOW MACH N
10. =,F9.6,/,20X,7HLHAT = ,11.5X,7HMHAT = ,11.5X,7HNHAT = ,11.22X.
29HOPTION = ,12,/,20X,8HGAMMA = ,F9.6,///)
220 FORMAT(40H THE ABSORBER HAS THE FOLLOWING GEOMETRY ,///)
230 FORMAT (20X,20HTHE LINER STARTS AT ,F9.6,13H AND ENDS AT ,F9.6,5X,
117HAPERTURE WIDTH = ,F9.6,/)
235 FORMAT (20X,8HBETAL = ,F9.6,3X,F9.6,///)
240 FORMAT (20X,23HTHE BACKING DISTANCE = ,F9.6,24X,15HTHE APERTURE ME
1AN FLOW = ,F9.6,/,20X,18HAPERTURE LENGTH = ,F9.6,28X,19HEFFECTIVE
2 LENGTH = ,F9.6,/,20X,10HEPSILON = ,F9.6,36X,14HCAVITY WIDTH = ,
3 F9.6,/,20X,16HCAVITY DENSITY = ,F9.6,30X,18HAPERTURE DENSITY =
4 ,F9.6,/,20X,20HCAVITY SOUND SPEED = ,F9.6,///)
250 FORMAT (39H THE INJECTOR RESPONSE IS DESCRIBED BY ,///)
260 FORMAT (20X,8HBETAI = ,F9.6,3X,F9.6,///)
270 FORMAT (20X,40HFOR THE CROCCO SENSITIVE TIME LAG THEORY,16X,4HN =
1,F9.6,4X,6HTAU = ,F9.6,/,
2 ,20X,16HINITIAL BETAI = ,F9.6,3X,F9.6,///)
280 FORMAT (20X,80HFOR THIS OPTION FREQUENCY IS INPUT AND RPTAI IS CAL
1CULATED. INPUT FREQUENCY IS ,F9.6,3X,F9.6,/,20X,6HWWO = ,F9.6,3X,
2,F9.6,/)
285 FORMAT (20X,27HTHE FIRST GUESS OF BETAI IS,F9.6,3X,F9.6,///)
290 FORMAT (37H THE NOZZLE RESPONSE IS DESCRIBED BY ,///)
300 FORMAT (20X,43HTHE FUNDAMENTAL FREQUENCY FOR THIS MODF IS ,F9.6,3X
1,3H0.0,/)
305 FORMAT (20X,27HTHE FIRST GUESS OF OMEGA IS,F9.6,3X,F9.6,///)
310 FORMAT (20X,8HBETAN = ,F9.6,3X,F9.6,///)
320 FORMAT (43H MISCELLANEOUS INFORMATION FOR THIS PROBLEM,///,20X,26H
1NUMBER OF TERMS RADIAL ,12,4X,13HLONGITUDINAL ,12,9X,8HIDMAX =
2,14,/,20X,8HERROR = ,F9.6,/)
350 FORMAT (78H ITER REAL OMEGA IMAG OMEGA ERRORS
1 MODULUS ANGLE,/)
360 FORMAT (78H ITER REAL BETAI IMAG BETAI ERRORS
1 MODULUS ANGLE,/)
365 FORMAT (1X,51H***THIS PROBLEM DOES NOT APPEAR TO BE CONVERGING***)
370 FORMAT (27H THE EQUIVALENT BETAI IS ,F9.6,3X,F9.6,///)
380 FORMAT (21H THE FINAL BETAI IS,F9.6,3X,F9.6,///)
385 FORMAT (28H N AND TAU CONVERGED FOR RN=,F9.6,/,
1 5X,29HTHE CALCULATED N AND TAU ARE ,F9.6,5H AND ,F9.6,16H
2RESPECTIVELY,/,5X,18HGIVING A BETAI OF ,F9.6,3X,F9.6,///)
386 FORMAT (30H NO AND TAUO CONVERGED FOR RN=,F9.6,/,
1 5X,31HTHE CALCULATED NO AND TAUO ARE ,F9.6,5H AND ,F9.6,16H
2RESPECTIVELY,/,5X,18HGIVING A BETAI OF ,F9.6,3X,F9.6,///)

```

ORIGINAL PAGE IS  
OF POOR QUALITY

```
390 FORMAT (/ .44H***** THIS PROBLEM HAS NOT CONVERGED *****//)
501 FORMAT (26H THE SLOT IMPEDANCES ARE,
1
2 //)
10X,3HRO=,G13.6,5X,3HKI=,G13.6,
666 FORMAT (2X,13,5X,G13.6,2X,G13.6,15X,F10.6,5X,F10.6)
900 FORMAT (5X,37HYOUR VALUE FOR MACH IS EXTREMELY HIGH,G10.4,/)
902 FORMAT (5X,39HYOUR VALUE FOR LFNGTH IS EXTREMELY HIGH,G10.4,/)
904 FORMAT (5X,42HYOUR PRODUCT OF WA*BETAL IS EXTREMELY HIGH,2G10.4,/)
906 FORMAT (5X,38HYOUR VALUE FOR BETAN IS EXTREMELY HIGH,2G10.4,/)
908 FORMAT (5X,35HYOUR VALUE FOR BR IS EXTREMELY HIGH,G10.4,/)
910 FORMAT (5X,36HYOUR VALUE FOR AMF IS EXTREMELY HIGH,G10.4,/)
912 FORMAT (5X,35HYOUR VALUE FOR IN IS EXTREMELY HIGH,G10.4,/)
914 FORMAT (5X,36HYOUR VALUE FOR TAU IS EXTREMELY HIGH,G10.4,/)
916 FORMAT (5X,55HTHE MAGNITUDE OF YOUR VALUE FOR BETAI IS EXTREMELY H
1IGH,2G10.4,/)
920 FORMAT (5X,37HYOUR VALUE FOR GAMMA IS EXTREMELY LOW,G10.4,/)
922 FORMAT (5X,38HYOUR VALUE FOR LENGTH IS EXTREMELY LOW,G10.4,/)
924 FORMAT (5X,34HYOUR VALUE FOR BR IS EXTREMELY LOW,G10.4,/)
926 FORMAT (5X,35HYOUR VALUE FOR AMF IS EXTREMELY LOW,G10.4,/)
930 FORMAT (5X,109HWITH THIS APEKTURE THE INTEGRATION OVER THE LINFR S
1HOULD BE CARRIED OUT SEE COMMENT CARDS ABOVE 500 CONTINUE,G10.4,
2//)
940 FORMAT (5X,89HYOU HAVE NOT KEPT ENOUGH TERMS IN THE LONGITUDINAL D
1IRECTION FOR THE MODE YOU HAVE CHOSEN,/,6HNLS = ,I2,5X,7HNHAT = ,
2I2,/)
942 FORMAT (5X,84HYOU HAVE NOT KEPT ENOUGH TERMS IN THE RADIAL DIRECTI
1ON FOR THE MODE YOU HAVE CHOSEN,/,5X,6HLTS = ,I2,5X,7HLHAT = ,I2,
2//)
950 FORMAT (5X,61HTHIS FREQUENCY IS EXTREMELY HIGH FOR THE MODE YOU HA
1VE CHOSEN,/)
960 FORMAT (5X,98HONE OF THE VALUES LISTED BELOW IS NOT PHYSICALLY MEA
1NINGFUL THIS SET OF DATA WILL NOT BE EXECUTED,/,5X,7HMACH = ,G10.
24,5X,8HGAMMA = ,G10.4,5X,9HLENGTH = ,G10.4,5X,8HBETAL = ,2G10.4,/,
35X,8HBETAN = ,2G10.4,5X,5HXA = ,G10.4,5X,5HXB = ,G10.4,5X,5HIN = ,
4G10.4,5X,6HTAU = ,G10.4,////)
980 FORMAT (5X,70HTHE ACOUSTIC MODE YOU SELECTED IS TOO HIGH FOR THIS
1PROGRAM TO HANDLE,/,5X,7HLHAT = ,I2,5X,7HMHAT = ,I2,5X,7HNHAT = ,
2I2)
END
```



```
C* FUNCTION BESRT(M,L)
C* THIS FUNCTION SUBROUTINE IS A TABLE OF THE ROOTS OF THE BESSEL
C* FUNCTION OF THE FIRST KIND.
DIMENSION A(4,10)
DIMENSION B(1,10)
IF (M.EQ.0) GO TO 10
A(1,1) = 1.84118378
A(1,2) = 5.33144277
A(1,3) = 8.53631637
A(1,4) = 11.7060049
A(1,5) = 14.86358863
A(1,6) = 18.01552786
A(1,7) = 21.16436986
A(1,8) = 24.31132666
A(1,9) = 27.45705057
A(1,10) = 30.60192297
A(2,1) = 3.05423693
A(2,2) = 6.70613319
A(2,3) = 9.96946782
A(2,4) = 13.17037086
A(2,5) = 16.34752232
A(2,6) = 19.51291278
A(2,7) = 22.67158177
A(2,8) = 25.82603714
A(2,9) = 28.97767277
A(2,10) = 32.12732702
A(3,1) = 4.20118894
A(3,2) = 8.01523660
A(3,3) = 11.34592431
A(3,4) = 14.58584829
A(3,5) = 17.78874707
A(3,6) = 20.97247624
A(3,7) = 24.14489743
A(3,8) = 27.31005793
A(3,9) = 30.47026881
A(3,10) = 33.62694918
A(4,1) = 5.31755313
A(4,2) = 9.28239629
A(4,3) = 12.68190844
A(4,4) = 15.96410704
A(4,5) = 19.19602680
A(4,6) = 22.40103227
A(4,7) = 25.58975968
A(4,8) = 28.76783622
A(4,9) = 31.93853934
A(4,10) = 35.10391668
BESRT=A(M,L)
RETURN
10 CONTINUE
B(1,1) = 0.00000000
B(1,2) = 3.83170597
B(1,3) = 7.01558667
B(1,4) = 10.17346814
B(1,5) = 13.32369194
B(1,6) = 16.47063005
B(1,7) = 19.61585851
B(1,8) = 22.76008438
B(1,9) = 25.90367209
B(1,10) = 29.04682853
BESRT=B(1,L)
RETURN
END
```

```
C* FUNCTION BESVL(M,L)
C* THIS FUNCTION SUBROUTINE IS A TABLE OF THE VALUES OF THE BESSEL
C* FUNCTION OF THE FIRST KIND.
DIMENSION A(4,10)
DIMENSION C(1,10)
IF (M.EQ.0) GO TO 10
A(1,1) = 0.58186522
A(1,2) = -0.34612620
A(1,3) = 0.27329994
A(1,4) = -0.23330442
A(1,5) = 0.20701265
A(1,6) = -0.18801749
A(1,7) = 0.17345905
A(1,8) = -0.16183821
A(1,9) = 0.15228207
A(1,10) = -0.14424290
A(2,1) = 0.48649868
A(2,2) = -0.31353045
A(2,3) = 0.25474416
A(2,4) = -0.22088158
A(2,5) = 0.19793743
A(2,6) = -0.18101000
A(2,7) = 0.16783553
A(2,8) = -0.15719517
A(2,9) = 0.14836378
A(2,10) = -0.14087833
A(3,1) = 0.4343942763
A(3,2) = -0.2911584413
A(3,3) = 0.240738175
A(3,4) = -0.210965204
A(3,5) = 0.190419022
A(3,6) = -0.175048405
A(3,7) = 0.162954965
A(3,8) = -0.153102409
A(3,9) = 0.144866574
A(3,10) = -0.137844513
A(4,1) = 0.3996514545
A(4,2) = -0.2743809949
A(4,3) = 0.229590468
A(4,4) = -0.202763849
A(4,5) = 0.184029896
A(4,6) = -0.169878516
A(4,7) = 0.158655372
A(4,8) = -0.149451156
A(4,9) = 0.141714307
A(4,10) = -0.135086328
RESVL = A(M,L)
RETURN
10 CONTINUE
C(1,1) = 1.00000000
C(1,2) = -0.4027588095
C(1,3) = 0.301128303
C(1,4) = -0.249704877
C(1,5) = 0.218359407
C(1,6) = -0.19645371
C(1,7) = 0.180063375
C(1,8) = -0.167184600
C(1,9) = 0.156724985
C(1,10) = -0.148011108
RESVL = C(1,L)
RETURN
END
```

```
      SUBROUTINE LINER(XA,XB,PIL,L,N,MHAT,MUX,CIOM,MACH,BLINER,LTS,
1     INTS,CZERO)
C* THIS SUBROUTINE IS USED TO CARRY OUT THE INTEGRATION OVER THE
C* LINER
      REAL NMNP,NPNP,MACH
      COMPLEX MUX,CIOM,BLINER,SUM1,SUM2,CZERO
      DIMENSION MUX(50,10)
      BLINER= CZERO
      DO 47 LP=1,LTS
      SUM1= CZERO
      SUM2= CZERO
      DO 45 N1=1,NTS
      NP= N1 - 1
      RN= FLOAT(NP)
      RNPIL= RN*PIL
      IF (N.EQ.NP) GO TO 42
      NMNP= FLOAT(NP - N)*PIL
      NPNP= FLOAT(NP + N)*PIL
      SUM1= SUM1 + MUX(N1,LP)*(SIN(NMNP*XB)/2./NMNP + SIN(NPNP*XB)/2./
1     NPNP - SIN(NMNP*XA)/2./NMNP - SIN(NPNP*XA)/2./NPNP)
      SUM2= SUM2 + MUX(N1,LP)*RNPIL*(COS(NMNP*XB)/2./NMNP + COS(NPNP*
1     XB)/2./NPNP - COS(NMNP*XA)/2./NMNP - COS(NPNP*XA)/2./NPNP)
      GO TO 45
42  CONTINUE
      IF (NP.EQ.0) GO TO 44
      SUM1= SUM1 + MUX(N1,LP)*((XB - XA)/2. + (SIN(2.*RNPIL*XB) -
1     SIN(2.*RNPIL*XA))/(4.*RNPIL))
      SUM2= SUM2 - MUX(N1,LP)*RNPIL*(SIN(RNPIL*XB)**2 - SIN(RNPIL*XA)*
1     **2)/(2.*RNPIL)
      GO TO 45
44  CONTINUE
      SUM1= MUX(N1,LP)*(XB - XA)
45  CONTINUE
      BLINER= BLINER - BESVL(MHAT,LP)*(CIOM*SUM1 + MACH*SUM2)
47  CONTINUE
      RETURN
      END
```