

2mif

**NASA TECHNICAL
MEMORANDUM**

NASA TM X-71973
COPY NO.

NASA TM X-71973

A Recurrence Matrix Method for the Analysis of Longitudinal
and Torsional Vibrations in Non-Uniform Multibranch beams
with Variable Boundary Conditions

(NASA-TM-X-71973) A RECURRENCE MATRIX
METHOD FOR THE ANALYSIS OF LONGITUDINAL
AND TORSIONAL VIBRATIONS IN NON-UNIFORM
MULTIBRANCH BEAMS WITH VARIABLE BOUNDARY

(NASA) -152 p HC \$5.00

IS7

CSCL 20K

N74-26365

Unclass

G3/32 40523

Robert B. Davis and Maria V. Stephens

May 6, 1974



This informal documentation medium is used to provide accelerated or special release of technical information to selected users. The contents may not meet NASA formal editing and publication standards, may be revised, or may be incorporated in another publication.

**NATIONAL AERONAUTICS AND SPACE ADMINISTRATION
LANGLEY RESEARCH CENTER, HAMPTON, VIRGINIA 23665**

1. Report No. TM X-71973	2. Government Accession No.	3. Recipient's Catalog No.	
4. Title and Subtitle A Recurrence Matrix Method for the Analysis of Longitudinal and Torsional Vibrations in Non-Uniform Multibranch Beams with Variable Boundary Conditions		5. Report Date May 6, 1974	
7. Author(s) Robert B. Davis Maria V. Stephens		6. Performing Organization Code 55.110	
9. Performing Organization Name and Address NASA Langley Research Center Hampton, VA 23665		8. Performing Organization Report No. 10. Work Unit No. 760-62-01-23	
12. Sponsoring Agency Name and Address National Aeronautics & Space Administration Washington, DC 20546		11. Contract or Grant No. 13. Type of Report and Period Covered TMX 1973-1974	
15. Supplementary Notes			
16. Abstract An approximate method for calculating the longitudinal and torsional natural frequencies and associated modal data of a beamlike, variable cross section multibranch structure is presented in this paper. The procedure described in this paper is the numerical integration of the first order differential equations that characterize the beam element in longitudinal motion and that satisfy the appropriate boundary conditions.			
17. Key Words (Suggested by Author(s)) (STAR category underlined) Vibrations, Longitudinal, Torsional, Multibranch		18. Distribution Statement	
19. Security Classif. (of this report) Unclassified	20. Security Classif. (of this page) Unclassified	21. No. of Pages 152	22. Price* \$5.00

*Available from { The National Technical Information Service, Springfield, Virginia 22151

{ STIF/NASA Scientific and Technical Information Facility, P.O. Box 33, College Park, MD 20740

TABLE OF CONTENTS

	Page
SUMMARY	1
INTRODUCTION	2
SYMBOLS	4
ANALYSIS	6
Derivation of Equations for a Single Beam Vibrating in the Axial Direction	7
Method of Integration	10
Derivation of Member Influence Matrix	11
Derivation of the Boundary Equations of a Joint Having From One to Four Structural Members	14
Equilibrium Equations	14
Compatibility Equations	15
Calculation of the Natural Frequencies	19
Application of the Method to Torsional Vibrations	21
NUMERICAL EXAMPLE FOR IDEALIZED BEAM	24
Matrix Formulation for Idealized Beam	24
Calculation of the Modal Data	27
TYPICAL LAUNCH VEHICLE APPLICATION	28
DISCUSSION OF METHOD APPLICATION	29
Computational Accuracy	29
Advantages of the Recurrence Method for Longitudinal and Torsional Vibration Analysis	30
CONCLUDING REMARKS	32
REFERENCES	33

PRECEDING PAGE BLANK NOT FILMED

	Page
TABLE I - PHYSICAL CHARACTERISTICS OF LAUNCH VEHICLE	34
TABLE II - PHYSICAL CHARACTERISTICS OF PAYLOAD	40
TABLE III - A COMPARISON OF THE NATURAL LONGITUDINAL FREQUENCIES CALCULATED BY THE LUMPED MASS METHOD OF REFERENCE 1 AND THE FINITE DIFFERENCE PROCEDURE FOR THE ROCKET - VEHICLE CONFIGURATION	40
Figure 1. - Numerical example natural longitudinal mode shapes	41
Figure 2. - Numerical example natural torsional mode slopes. . .	42
Figure 3. - Calcomp plotting instruction card	43
Figure 4. - Numerical example plot of torsional modes	44
Figure 5. - Numerical example plot of torsional mode slopes . . .	45
Figure 6. - Numerical example plot of torque distribution for first mode	46
Figure 7. - Numerical example plot of torque distribution for second mode	47
Figure 8. - Numerical example plot of torque distribution for third mode	48
Figure 9. - Rocket-vehicle mass per inch. m values from table I and II	49
Figure 10.- Rocket-vehicle axial extension coefficient AE values from tables I and II	50
Figure 11.- Rocket-vehicle longitudinal natural mode shapes . . .	51
Figure 12.- Frequency percent error as a total number of stations	52
Figure 13.- Computer program flow diagram.	53
APENDIX A - DESCRIPTION OF COMPUTER PROGRAM	54
METHOD OF INPUT	55
SYMBOLS	58
Input Namelist Names	58
NOMENCLATURE	59

	Page
Input-Output Variable Names of Main Program	59
Input Plot Variable Names	64
OUTPUT	67
DIMENSIONALIZATION	68
DISCUSSION OF COMPUTER PROGRAM	69
* APPENDIX B - COMPUTER PROGRAM LISTING	78
APPENDIX C - COMPUTER PRINTOUT OF TORSIONAL MODAL DATA FOR NUMERICAL EXAMPLE	118

SUMMARY

An approximate method for calculating the longitudinal and torsional natural frequencies and associated modal data of a beamlike, variable cross section multibranch structure is presented in this paper.

Natural vibration characteristics of systems having discontinuous physical properties are usually most conveniently solved by means of the digital computer. The procedure described in this paper is the numerical integration of the first order differential equations that characterize the beam element in longitudinal motion and that satisfy the appropriate boundary conditions.

Numerical examples included in this paper are an application to a solid fuel launch vehicle and an idealized beam. A complete description and discussion of the computer program is also provided.

INTRODUCTION

It is often necessary to determine the longitudinal and/or torsional natural frequencies and related modal data of structures since they provide basic dynamic information. Typical structures which require knowledge of the longitudinal or torsional vibration characteristics include piles, turbines, machine shafts, bridges, and towers. Natural mode characteristics are also a valuable tool in analyzing the responses of a structure due to disturbing forces.

In all but the most elementary beams, approximate methods must be utilized to determine the longitudinal or torsional natural frequencies. There are numerous references of approximate methods to calculate the longitudinal natural frequencies for variable cross section beams (see references 1, 2, and 3). However, there is a scarcity of available references to calculate the longitudinal frequencies of branched beams; in particular when the branches are partially constrained to the main member.

The purpose of this paper is to present an approximate method for the calculation of the longitudinal natural frequencies and mode shapes for a variable cross section, multibranch beam. This method is derived for the general case in which four arbitrary members intersect at a joint. Also, the branches may be fully or partially constrained to the main member.

The calculation of the longitudinal natural frequencies for a variable cross section beam has been treated by both a stiffness and flexibility matrix approach (references 1 and 4). In the stiffness matrix approach, the beam is idealized as a number of point masses connected by springs. The simultaneous equations of harmonic motion for the point masses when written in matrix notation, yield an eigenvalue problem which is

solvable by standard methods.

One disadvantage of both the flexibility and stiffness matrix methods is that considerable time may be spent averaging the properties. Another disadvantage is that it may require the storage of very large matrices in the computer. In fact, for some cases, the number of masses required for accuracy may be greater than the computer storage capability.

The method presented in this paper utilizes a finite difference approach. The station properties are computed from one end of the beam to the other by numerical integration. This offers an important advantage for continuous structures. Through a recurrence equation, a very large number of stations may be utilized without storing a large matrix in the computer.

SYMBOLS

$A(x)$	stressed cross-sectional area, inch ² (meter ²)
A	matrix (see equation (10))
B	matrix (see equation (19))
$E(x)$	modulus of elasticity, pounds/inch ² (newton/meter ²)
$G(x)$	modulus of shear, pound/inch ² (newton/meter ²)
$I_p(x)$	polar mass moment of inertia, pound-second ² (newton - second ²)
$J(x)$	polar moment of inertia, inch ⁴ (meters ⁴)
L	overall length of main beam (see sketch 11)
$m(x)$	mass per unit length, $\frac{\text{pound} - \text{second}^2}{\text{inch}^2}$ ($\frac{\text{newton} - \text{second}^2}{\text{meter}^2}$)
N	total number of structural members
$P(x)$	axial force, pounds (newtons)
$T(x)$	torque, inch-pounds (meter-newtons)
t	time, sec.
U	modal displacement in the axial direction, inches (meters)
$U(s,n)$	modal deflection of the nth integration station of the sth member, inches (meters)
U''	modal acceleration in the axial direction, inches/second ² (meters/second ²)
W	matrix (see equation (8))
x	longitudinal coordinate along beam centerline, inches (meters)
x_n	longitudinal coordinate at the nth integration station, inches (meters)
Δx_n	increment in recurrence solution, $x_{n+1} - x_n$, inches (meters)
Y_n	matrix (see equation (15))
$Y(s,n)$	the state vector at the nth integration station of the sth member

$Y'(s,n)$	the X derivative of $Y(s,n)$, $\frac{dY}{dx}(s,n)$
Z	the combination of all the state vectors at the boundary
$\alpha(s)$	translational flexibility constant between members s and i , inch/pounds (meter/newtons)
$\beta(s)$	spring constant between member s and ground, pounds/inch (newtons/meter)
η	constant (see equation (54))
$\gamma(\omega)$	matrix (see equation (37))
$\zeta(x)$	amplitude of mode shape, unitless
$\theta(x,t)$	cross-section rotation (radians)
ω	circular frequency

Subscripts

i	index denoting the i th beam or branch
j	index denoting the j th beam or branch
k	index denoting the k th branch
ℓ	index denoting the ℓ th branch
s	index denoting a general beam or branch
$r(i)$	rth station of the i th beam or branch
$r(j)$	rth station of the j th beam or branch
$r(k)$	rth station of the k th branch
$r(\ell)$	rth station of the ℓ th branch
$r(s)$	rth station of the s th beam or branch

Matrix notation

{ }	column matrix
[]	square or rectangular matrix
[]	row matrix
[1]	identity matrix

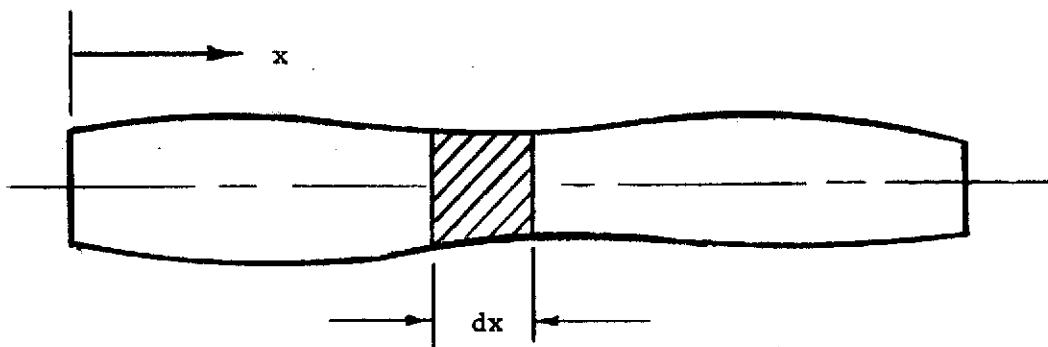
ANALYSIS

Two first order differential equations are developed for a beam element in longitudinal motion. The two equations are the elastic equation which relates the displacement function to the axial force and axial stiffness and the dynamic equation (D'Alembert's Principle) in which the inertial forces are equated to the applied forces. These equations can be put in matrix notation and integrated along the beam by a numerical procedure.

Each end of a beam is considered a joint. A method is developed for determining the boundary value equations at each joint. By combining all boundary value equations into products of a coefficient and station property matrix, it is demonstrated how the natural frequencies and corresponding mode shapes can be calculated. The derivations are based on one dimensional beam theory; therefore to apply this method to structures for which one dimensional beam theory is not applicable, one must proceed with reservations.

Derivation of Equations for a Single Beam

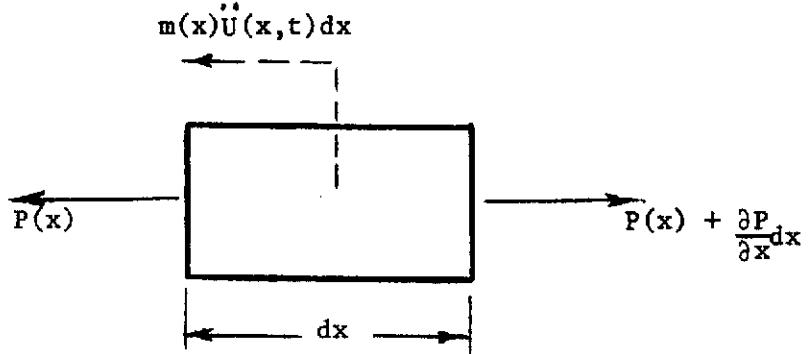
Undergoing Longitudinal Motion



Variable cross section beam.

Sketch 1.

Consider a general non-uniform beam of sketch 1 which is vibrating in the longitudinal direction. If an element of width dx is removed from the beam then we have the following inertial and applied forces of sketch 2.



Beam element in longitudinal motion.

Sketch 2.

Summing forces we get

$$P(x) + \frac{\partial P(x)}{\partial x} dx - P(x) - m(x) \ddot{U}(x,t) dx = 0 \quad (1)$$

$$\frac{\partial P(x)}{\partial x} = m(x) \ddot{U}(x,t) \quad (2)$$

Assuming that the element vibrates in simple harmonic motion of frequency ω and amplitude $\zeta(x)$, then, $U(x,t) = \zeta(x)e^{i\omega t}$ or

$$\ddot{U}(x,t) = -\omega^2 U(x,t) \quad (3)$$

Substituting equation (3) into equation (2), the equation of longitudinal motion for the beam element becomes

$$\frac{\partial P}{\partial x}(x) = -m(x) \omega^2 U(x,t) \quad (4)$$

From elementary beam theory the equation relating the displacement function to the axial force and axial stiffness is

$$\frac{\partial U}{\partial x}(x) = \frac{P(x)}{A(x)E(x)} \quad (5)$$

If the beam is assumed to vibrate longitudinally in simple harmonic motion then we can characterize the axial force and displacement as

$$P(x,t) = P(x)e^{i\omega t} \quad (6)$$

$$U(x,t) = U(x)e^{i\omega t}$$

where $i = \sqrt{-1}$

Equation (4) and (5) can be written in matrix notation as

$$\frac{d}{dx} \begin{Bmatrix} P \\ U \end{Bmatrix} = \begin{bmatrix} 0 & -m\omega^2 \\ 1/AE & 0 \end{bmatrix} \begin{Bmatrix} P \\ U \end{Bmatrix} \quad (7)$$

or

$$\{Y'(x)\} = [W(x)]\{Y(x)\} \quad (8)$$

$$\text{where } Y' = \frac{dY(x)}{dx} \text{ and } Y(x) = \begin{Bmatrix} P \\ U \end{Bmatrix} \quad (9)$$

Method of Integration

We can integrate this matrix along the beam by a variety of numerical procedures. For example, the second order Runge-Kutta integration of this matrix is given by the following

$$\{Y_{n+1}\} = \{Y_n\} + \frac{1}{2} [\{k_1\} + \{k_2\}] \quad (10)$$

where $\{Y_n\} = \{Y(x_n)\}$ and $\Delta x_n = x_{n+1} - x_n$

$$\{k_1\} = \Delta x_n [w_n] \{Y_n\} \quad (11)$$

$$\{k_2\} = \Delta x_n [w_{n+1}] [\{Y_n\} + \{k_1\}] \quad (12)$$

Replacing the k vectors leads to

$$\{Y_{n+1}\} = [A] \{Y_n\} \quad (13)$$

The transfer matrix has the form

$$[A_n] = [-1] + \frac{\Delta x_n}{2} [[w_{n+1}] + [w_n]] + \frac{\Delta x_n^2}{2} [w_{n+1}] [w_n] \quad (14)$$

Combining these matrices, A becomes

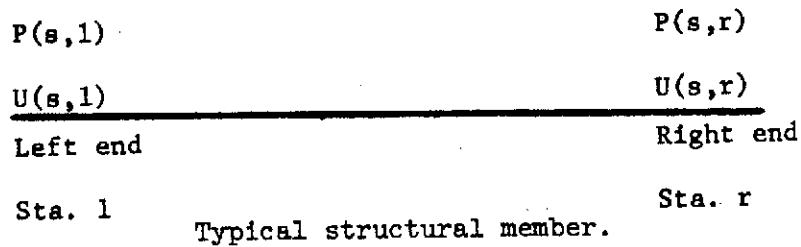
$$[A] = \begin{bmatrix} 1 - \frac{m_{n+1}\omega^2}{2 A_n E_n} \Delta x_n^2 & \frac{-(m_{n+1} + m_n) \Delta x_n \omega^2}{2} \\ \frac{\Delta x_n}{2} \left(\frac{1}{A_{n+1} E_{n+1}} + \frac{1}{A_n E_n} \right) & 1 - \frac{m_n \omega^2}{2 A_{n+1} E_{n+1}} \Delta x_n^2 \end{bmatrix} \quad (15)$$

Therefore (13) may be written as:

$$\{Y_{n+1}\} = \begin{bmatrix} a_{11} & a_{12} \\ a_{21} & a_{22} \end{bmatrix} \{Y_n\} \quad (16)$$

Derivation of Member Influence Matrix

The member influence matrix relates the right end boundary conditions to the left end boundary conditions. For example, in sketch 3, the left end boundary conditions are $P(s,1)$ and $U(s,1)$ and the right end boundary conditions are $P(s,r(s))$ and $U(s,r(s))$.



Sketch 3.

The right end boundary conditions can be related to the left boundary conditions by the influence matrix $[B]$ and takes the form of

$$\begin{Bmatrix} P(s,r(s)) \\ U(s,r(s)) \end{Bmatrix} = [B(s)] \begin{Bmatrix} P(s,1) \\ U(s,1) \end{Bmatrix} \quad (17)$$

The member influence matrix $[B(s)]$ is computed by means of successive multiplication of the previously developed interval transfer matrix, $[A(s)]$. For the typical member of sketch 3

$$\begin{Bmatrix} P(s,n+1) \\ U(s,n+1) \end{Bmatrix} = [A(s,n)] \begin{Bmatrix} P(s,n) \\ U(s,n) \end{Bmatrix} \quad (18)$$

Each interval transfer matrix relates the station properties at the two ends of an interval. For example, again referring to sketch 3

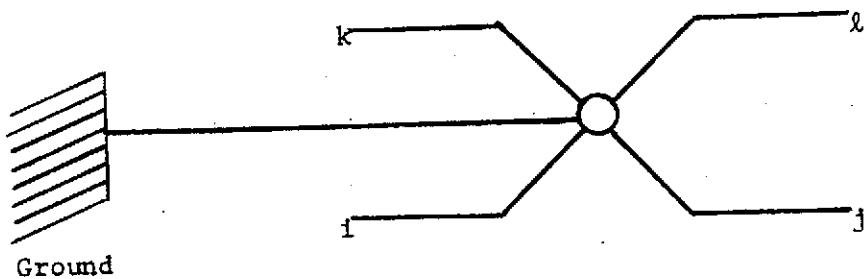
$$[B(s)] = [A(s,(r(s)-1))] [A(s,(r(s)-2))] \dots [A(s,1)] \quad (19)$$

Elements of the interval transfer matrix are functions of the frequency and physical characteristics which mark the boundary of the interval. Equation (17) can now be written in terms of the state vector for the s th beam with integrating stations $n = 1$ to $n = r(s)$.

$$\{Y(s,(r(s))\} = [B(s)] \{Y(s,1)\} \quad (20)$$

Derivation of the Boundary Equations of a Joint

Having from One to Four Structural Members



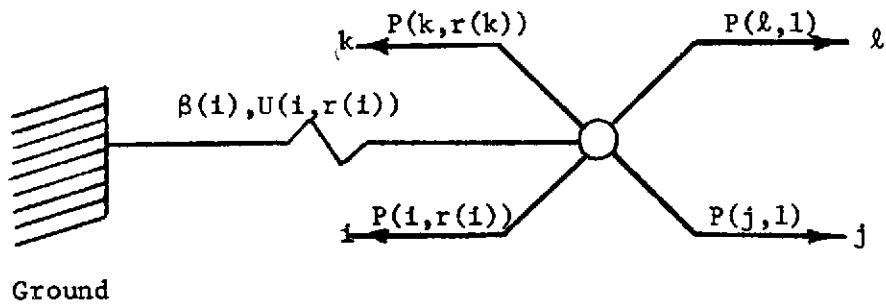
Typical joint of a multibranch beam.

Sketch 4.

A multibranch beam can be considered as a truss-like structure which consists of several members tied together by a series of joints. At each joint the boundary values must satisfy a set of equilibrium and compatibility equations.

A general set of boundary equations is derived for four arbitrary members which are designated by the indices i , j , k , and ℓ (see sketch 4). The joint representation allows for a general flexibility constraint between the members i or j and each of the other members. Also included are provisions for a general elastic constraint between members i or j and an external ground. A ground constraint can be placed on member j only when member i does not exist.

Equilibrium equation.- If the joint in sketch 4 is removed, and a freebody diagram is made of the axial forces



Freebody diagram of axial forces on joint.

Sketch 5.

The equilibrium equation for all the forces acting in the longitudinal direction is

$$P(i,r(i)) + P(k,r(k)) + \beta(i) U(i,r(i)) - P(l,l) - P(j,l) = 0 \quad (21)$$

Equation (21) is derived on the assumption that four members are present, however this equation is valid for less than four members at the joint. Also the joint may be free of the ground constraint by setting $\beta(i)$ equal to zero. Then equation (21) may be rewritten as

$$P(i,r(i)) + \delta(k) P(k,r(k)) + \beta(i) U(i,r(i)) - \delta(l) P(l,l) - \delta(j) P(j,l) = 0 \quad (22)$$

where $\delta(\) = 0$ for $(\) = 0$

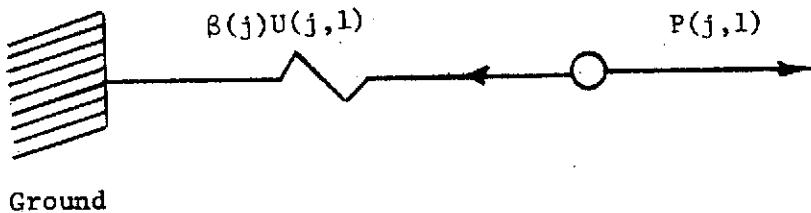
$\delta(\) = 1$ for $(\) \neq 0$

and i, j, k , or $l = 0$ if not present at joint

In the special case where member i is not present then the equilibrium equation is

$$\beta(j)U(j,1) - P(j,1) = 0 \quad (23)$$

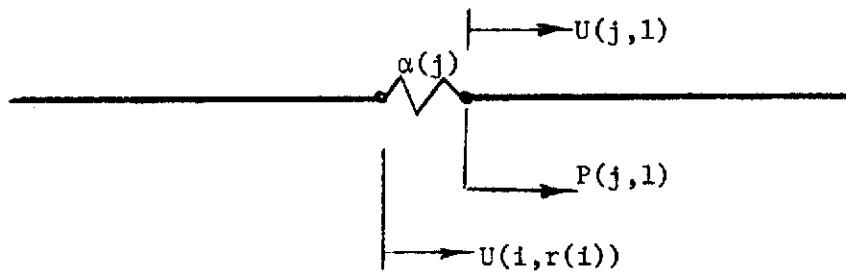
A freebody diagram of the axial forces on the joint of sketch 4 when members i, k , and l are absent is illustrated in sketch 6.



Freebody diagram of axial forces on joint with member i missing.

Sketch 6.

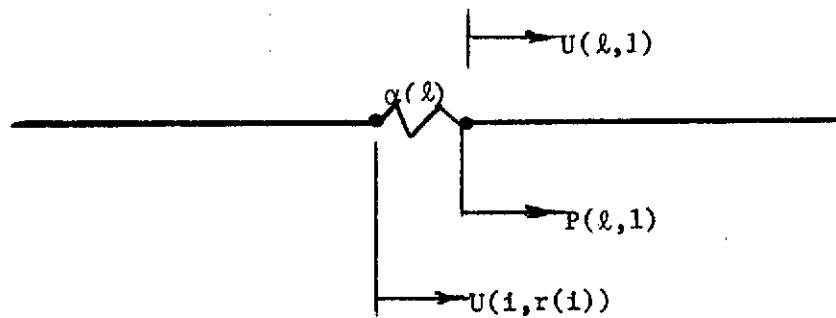
Compatibility equations.- There is a compatibility relationship between the main member (i or j) and each branch member. The compatibility equations are derived on the assumption that there is no relative displacement between members at a joint except through spring deformations.



Compatibility relationship with member i and j present.

Sketch 7.

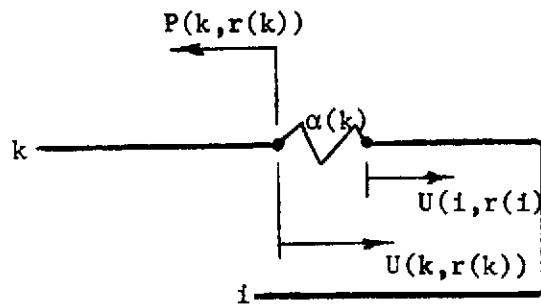
$$U(i, r(i)) + \alpha(j) P(j,1) - U(j,1) = 0 \quad (24)$$



Compatibility relationship with members i and l present.

Sketch 8.

$$U(i, r(i)) + \alpha(\ell) P(\ell, 1) - U(\ell, 1) = 0 \quad (25)$$



Compatibility relationship with members i and k present.

Sketch 9.

$$U(i, r(i)) - \alpha(k)P(k, r(k)) - U(k, r(k)) \quad (26)$$

The state vector (equation 9) can be put in the form

$$\{Y(i, r(s))\} = \begin{Bmatrix} P(i, r(s)) \\ U(i, r(s)) \end{Bmatrix} \quad (27)$$

Utilizing (27), the equilibrium equations (22) and (23) can be combined to give

$$\begin{aligned} & \delta(i)[1 \ \beta(i)]\{Y(i,r(i))\} - \delta(j)[1 \ (\delta(i) - 1) \ \beta(j)]\{Y(j,1)\} \\ & + \delta(i) \ \delta(k)[1 \ 0]\{Y(k,r(k))\} - \delta(i) \ \delta(l)[1 \ 0]\{Y(l,1)\} = 0 \end{aligned} \quad (28)$$

The three compatibility equations, (24) through (26), can be written as a function of the state vector.

$$\delta(i)\delta(j)[0 \ 1]\{Y(i,r(i))\} - \delta(i)\delta(j)[-a(j) \ 1]\{Y(j,1)\} = 0 \quad (29)$$

$$\delta(i)\delta(k)[0 \ 1]\{Y(i,r(i))\} - \delta(i)\delta(k)[a(k) \ 1]\{Y(k,r(k))\} = 0 \quad (30)$$

$$\delta(i)\delta(l)[0 \ 1]\{Y(i,r(i))\} - \delta(i)\delta(l)[-a(l) \ 1]\{Y(l,1)\} = 0 \quad (31)$$

From equation (20), we have the relationship for the s th beam at the last integration station to the s th beam at the first integration station.

$$\{Y(s,r(s))\} = [B(s)]\{Y(s,1)\} \quad (32)$$

By substituting equation (32) into equations (28), thru (31) the final form of the four boundary value equations is obtained.

$$\begin{aligned} & \delta(i)[1 \ B(i)][B(i)]\{Y(i,1)\} - \delta(j)[1 \ (\delta(i) - 1) \ B(j)][Y(j,1)\} \\ & + \delta(i)\delta(k)[1 \ 0][B(k)]\{Y(k,1)\} - \delta(i)\delta(l)[1 \ 0]\{Y(l,1)\} = 0 \end{aligned} \quad (33)$$

$$\delta(i)\delta(j)[0 \ 1][B(i)]\{Y(i,1)\} - \delta(i)\delta(j)[-a(j) \ 1]\{Y(j,1)\} = 0 \quad (34)$$

$$\delta(i)\delta(k)[0 \ 1][B(i)]\{Y(i,1)\} - \delta(i)\delta(k)[a(k) \ 1][B(k)]\{Y(k,1)\} = 0 \quad (35)$$

$$\delta(i)\delta(l)[0 \ 1][B(i)]\{Y(i,1)\} - \delta(i)\delta(l)[-a(l) \ 1]\{Y(l,1)\} = 0 \quad (36)$$

When applying these boundary value equations to a particular joint, any member and it's respective equation can be left out by setting it's index equal to zero. There are more unknowns than there are equations at each joint. The boundary values at each joint can only be found by solving all of the equations simultaneously. The complete set of boundary value equations for the structure can therefore be written collectively as a product of the beam boundary conditions column matrix and the coefficient square matrix.

$$[\gamma(\omega)]\{z\} = 0 \quad (37)$$

Calculation of the Natural Frequencies

The nontrivial solution to equation (37) requires the vanishing of the coefficient determinant

$$[\gamma(\omega)] = 0 \quad (38)$$

The expansion of the determinant of (38) yields the characteristic equation which is polynomial in ω . It is necessary to conduct a trial search by successive approximation of the eigenvalue ω to find the characteristic roots (or natural frequencies) which satisfy eq. (38).

For each characteristic root of equation (38) there is a corresponding eigenvector $\{Z\}$ where

$$\{Z\} = \begin{Bmatrix} Y(1,1) \\ \vdots \\ Y(N,1) \end{Bmatrix} \quad (39)$$

where

$$\{Y(s,n)\} = \begin{Bmatrix} P(s,n) \\ U(s,n) \end{Bmatrix} \quad (40)$$

The value of $\{Z\}$ can be determined by setting an arbitrary non-zero element of $Y(1,1)$, equal to unity and solving for the remaining elements in terms of the unit element from $\{Z\}$, creating $\{Z_{mod}\}$. The corresponding row is removed from $[Y(\omega)]$ the corresponding column is moved to the right side to convert (37) to the following

$$[Y_{mod}] \cdot \{Z_{mod}\} = -\{\text{column}\} \quad (41)$$

This set of simultaneous equations is solved for $\{Z_{mod}\}$. That element which was previously removed is then returned to $\{Z_{mod}\}$, forming the solution vector $\{Z\}$. After the left end boundary conditions of a beam are determined, then the $\{Y(s,n)\}$ modal data can be determined at each integration

station along a segment by successive use of the recurrence equation (13).

$$\{Y(s,n+1)\} = [A]\{Y(s,n)\} \quad (42)$$

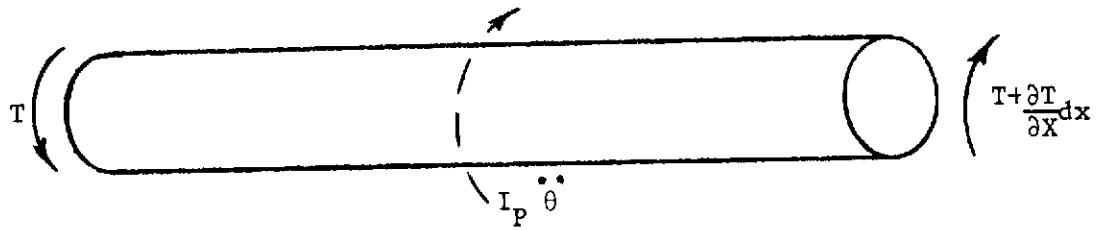
If the rate of change of the modal properties are desired, then from equation (8)

$$\{Y'(s,n)\} = [W]\{Y(s,n)\} \quad (43)$$

It should be noted that this method is applicable for determining the modal data corresponding to any frequency.

Application of the Method to Torsional Vibrations

If the elastic axis of all beams and branches are everywhere concentric, it is also possible to derive the two first order differential equations for torsional vibrations analogous to the longitudinal vibration equations.



Beam element in torsional motion.

Sketch 10.

By D'Alembert's Principle, the dynamic equation for torsional motion is found to be

$$-T(x) + T(x) + \frac{\partial T(x)}{\partial x} dx - I_p(x) \frac{\partial^2 \theta}{\partial t^2} dx = 0 \quad (44)$$

or

$$\frac{\partial T(x)}{\partial x} = I_p(x) \frac{\partial^2 \theta}{\partial t^2} \quad (45)$$

If simple harmonic motion is assumed, then

$$\ddot{\theta} = -\omega^2 \theta(x, t) \quad (46)$$

Substituting equation (46) into equation (45)

$$\frac{\partial T(x)}{\partial x} = -I_p(x) \omega^2 \theta(x, t) \quad (47)$$

For longitudinal vibrations, the corresponding analogy is

$$\frac{\partial P(x)}{\partial x} = -m(x) \omega^2 U(x,t) \quad (48)$$

The elastic equation for the cross-sectional rotational function for a concentric member is

$$\frac{\partial \theta(x)}{\partial x} = \frac{T(x)}{J(x)G(x)} \quad (49)$$

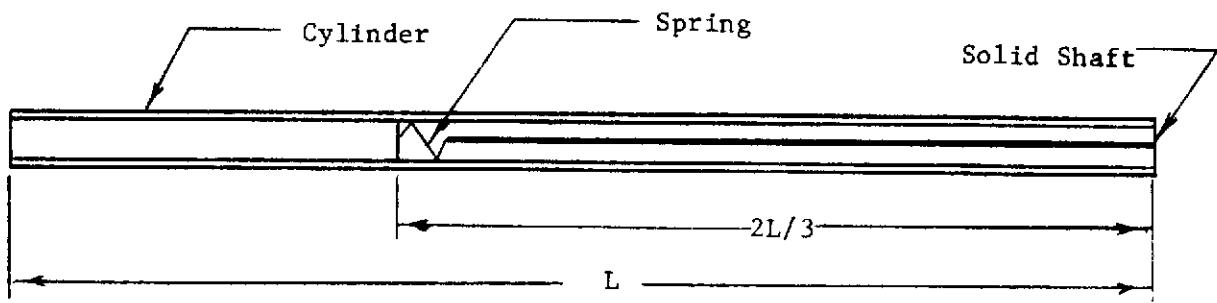
The corresponding equation for the longitudinal displacement function is

$$\frac{\partial U(x)}{\partial x} = \frac{P(x)}{A(x)E(x)} \quad (50)$$

If equation (48) is compared with equation (47) and equation (50) is compared with equation (49), it is seen that JG is analogous to AE , I_p is analogous to m , and θ is analogous to U . Therefore by replacing JG , I_p , and θ with AE , m , and U , respectively, all of the previously derived equations for the longitudinal vibrations of beams can be applied directly to torsional vibration analysis of beams whose elastic equation and equation of motion are characterized by equations (47) and (49).

NUMERICAL EXAMPLE FOR IDEALIZED BEAM

For the purpose of illustrating the procedure to calculate the longitudinal and torsional vibrations, an idealized beam is shown in sketch 11. This example consists of a solid uniform circular shaft attached by a spring to the inside of a cylindrical shell. The spring is assumed to elongate only in the longitudinal direction. The physical characteristics of the idealized beam are provided in the computer program discussion.



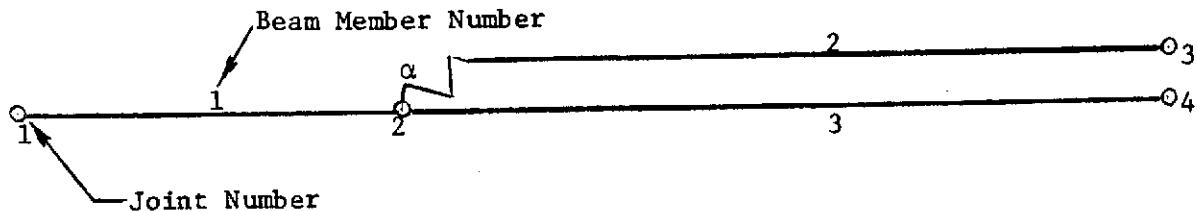
Idealized beam.

Sketch 11.

Matrix Formulation for Idealized Beam

The procedure, as outlined in the analysis, is to write the boundary value equations for each joint; put the equations in matrix notation, and arrange as a product of the coefficient matrix and station property matrix.

The mathematical model of the idealized beam is illustrated in sketch 12.



Numbering of joints and members.

Sketch 12.

$$[\gamma(\omega)]\{z\} = 0 \quad (50)$$

Beam Member

Joint Number	1	2	3	
(1)	$[1 \ 0]$	$[1 \ 0][B(1)]$	$[1 \ 0][B(2)]$	$-[1 \ 0]$
(2)	$[0 \ 1][B(1)]$	$-[-\alpha(\ell) \ 1][B(2)]$	$-[0 \ 1]$	$[P_1 \ U_1]$
(3)	$[0 \ 1][B(1)]$	$-[1 \ 0]$	$[1 \ 0][B(3)]$	$[P_2 \ U_2]$
(4)				$[P_3 \ U_3]$

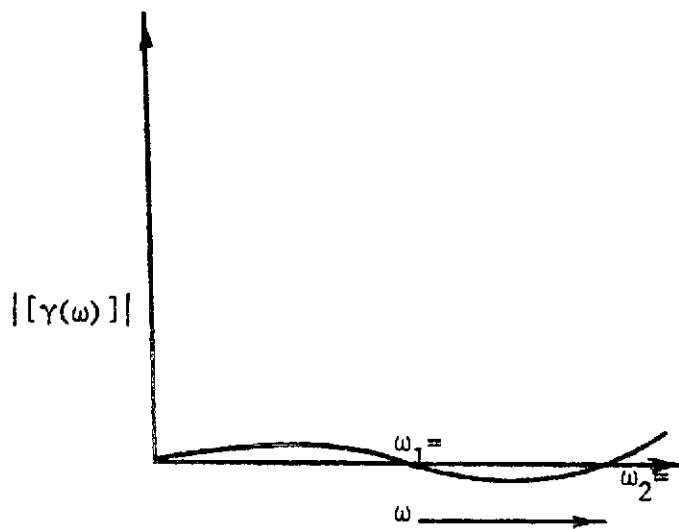
(51)

The vanishing of the coefficient matrix of eq. 51 is necessary to determine the natural frequencies, i.e.

$$[\gamma(\omega)] = 0 \quad (52)$$

$$\begin{bmatrix} -[1 & 0] & & \\ [1 & 0][B(1)] & [1 & 0][B(2)] & -[1 & 0] \\ [0 & 1][B(1)] & -[-\alpha(\ell) & 1][B(2)] & \\ [0 & 1][B(1)] & & -[0 & 1] \\ & -[1 & 0] & \\ & & [1 & 0][B(3)] \end{bmatrix} = 0 \quad (53)$$

Sketch (13) illustrates the plot of $|\gamma(\omega)|$ vs. ω . The natural frequencies are obtained when $|\gamma(\omega)| = 0$.



Natural frequencies of idealized beam.

Sketch 13.

Calculation of the Modal Data

The longitudinal mode shapes and natural frequencies are illustrated in figure 1. The first mode shape corresponds to the spring mode. The second mode shape corresponds to the first natural longitudinal frequency of the cylinder. The third mode shape corresponds to the first natural longitudinal frequency of the shaft, and the fourth mode shape corresponds to the second longitudinal frequency of the cylinder.

The mode shapes associated with the natural torsional frequencies of the idealized beam are illustrated in figure 2. The first mode shape corresponds to the first natural torsional frequency of the shaft. The second mode shape corresponds to the first natural torsional frequency of the cylinder. The third mode shape corresponds to the second natural torsional frequency of the shaft, and the fourth mode shape corresponds to the second natural torsional frequency of the cylinder.

In order to demonstrate the plotting capability of the program the Calcomp plotting instruction card is illustrated in figure 3. The modal data obtained from the Calcomp plot are illustrated in figures 4-8.

This simple numerical example demonstrates the versatility of the program by illustrating that it will calculate the spring modes, branch modes, and beam modes.

TYPICAL LAUNCH VEHICLE APPLICATION

In order to demonstrate the application of the matrix solution to a more realistic structure, a numerical example of an application to a solid fuel booster system is presented. The following assumptions apply to the analysis:

- (1) The solid-fuel mass is considered to adhere to the sides of the vehicle along the length and has no motion relative to the vehicle.
- (2) The contribution of the fuel stiffness to the vehicle axial stiffness is negligible.
- (3) Damping is considered to be negligible.
- (4) All deformations are one-dimensional and no consideration is given to bending or breathing effects of the cylindrical shell wall.

The vehicle physical characteristics were taken from reference 1 and are tabulated in tables I and II. These data are shown graphically in figures 9 and 10. The payload parameters were selected for a typical payload.

Utilizing the method outlined in this paper the natural mode shapes and frequencies were obtained. The longitudinal natural mode shapes for the vehicle and payload are given in figure 11. There is good correlation between the natural frequencies calculated by the lumped mass method of reference 1 and those calculated by the method discussed in this paper. The frequency comparisons are illustrated in table III. The finite difference method is believed to be more accurate for two reasons. The first reason is that there were approximately ten times as many integrating stations in the finite difference method as there were lumped masses. The other reason the finite difference approach should be more accurate is because the discontinuities were accurately input in the program.

DISCUSSION OF METHOD APPLICATION

Computational Accuracy

Some of the points of interest to the user in the application of this method are discussed in this section.

Choice of the number of integration stations.- Usually the choice of the number of stations will present no problem because linear interpolation between the input physical characteristics will provide the number of stations necessary for sufficient accuracy. The computer program developed for this analysis has an upper limit of 20 members (main beam and branches) and up to a total of 600 integrating stations. However, in order to optimize the computer processing time, consistent with the required accuracy it is not usually desirable to utilize the maximum number of stations available in the program.

In order to examine the accuracy of the method discussed, comparisons were made between exact and approximate solution for a beam of exponentially varying cross section. For the approximate solution, the beam was divided into n equally spaced intervals with the station properties input at each interval.

If the cross section varies as

$$A(x) = A_0 e^{\frac{2\eta x}{L}} \quad (54)$$

The theoretical solution for the natural longitudinal vibrations of the exponentially varying beam is derived in reference 1. The parameter used for frequency comparison (Appendix A of reference 1) is the percent error.

$$\text{Percent error} = \frac{\omega_{\text{exact}} - \omega_{\text{approx.}}}{\omega_{\text{exact}}} \times 100$$

The percent error for the first five elastic modes as a function of the total number of integration stations is given in figure 12 for the natural frequencies calculated by the method discussed in this paper vs. the theoretical natural frequencies. For this error analysis, the cross-sectional ratio of the beam ends is approximately 50. Even with this large variation of the cross-sectional area, it is found that the accuracy of the natural frequencies is primarily dependent on the number of integration stations rather than on the variation of the cross-sectional area. The first five natural longitudinal frequencies of the exponential beam were determined to be within one percent of their theoretical values when only 75 integrating stations were utilized. It has been found that a good rule of thumb for determining the minimum number of integration stations is $L/100$.

Advantages of the Recurrence Method for Longitudinal and Torsional Vibration Analysis

The prime advantage of the theoretical method over the lumped mass approach for determining the longitudinal vibration of beamlike structures is that the physical characteristics may be input directly, rather than resorting to a finite element representation and averaging the properties. Therefore, the input time is greatly reduced. Another important advantage is that the method is appropriate to highly discontinuous structures. Other advantages of the method are that it is appropriate to any boundary condition at the end point of a branch or beam and that a very large number

of stations may be utilized without the storage or inversion of large matrices.

CONCLUDING REMARKS

A finite difference method for the analysis of longitudinal and torsional vibrations of nonuniform multibranch beams is presented. The end of each beam or branch may be fully or partially constrained to the main member. The equations have been programmed for the CDC 6600 Series Computer Systems and have given excellent agreement when compared with numerical examples and exact solutions.

A numerical example of the procedure to calculate the longitudinal and torsional characteristics of an idealized beam along with an application of the method to a launch vehicle are provided. Comparisons of the method with exact solutions indicate that the accuracy of the solution is practically independent of cross-sectional variation, but is primarily dependent on the number of integration stations.

The primary advantage of this method is that highly discontinuous physical characteristics may be input directly. Another important advantage of the program is that the boundary condition may vary from fixed to free at each joint or end.

REFERENCES

1. Wingate, Robert T.: Matrix Analysis of Longitudinal and Torsional Vibrations in Nonuniform Multibranch Beams, NASA TN D-3844, Feb. 1967.
2. N. O. Myklestad: Fundamentals of Vibration Analysis. McGraw-Hill, New York, 1956.
3. W. T. Thompson: Matrix Solution for the Vibration of Non-Uniform Beams, J. Applied Mechanics.
4. J. S. Przemieniecka: Theory of Matrix Structural Analysis, McGraw-Hill, New York, 1968.
5. Harkins, Barbara; and Geiger, Thomas L.: Computer Program for Natural Modes and Frequencies of a Branched Beam in Transverse Vibration. ER-1441, Martin Company, Baltimore, Maryland.
6. Timoshenko, S.; and Young, D. H.: Vibration Problems in Engineering. Third ed., D. Van Nostrand Co. Inc., C-1955.

TABLE I
PHYSICAL CHARACTERISTICS OF LAUNCH VEHICLE

x, in.	x, meter	$\frac{m}{lb \cdot sec^2}$ / in^2	$\frac{m}{Newton \cdot sec^2}$ / $meter^2$	AE 1b $\times 10^{-6}$	AE Newton $\times 10^{-1}$
0.000	0.0000	0.036808	253.782	160.00	71.168
15.800	0.4013	0.046624	321.461	295.00	131.266
15.800	0.4013	0.046624	321.461	295.00	131.266
17.276	0.4388	0.046624	321.461	295.00	131.266
17.276	0.4388	0.046624	321.461	280.30	124.683
22.270	0.5658	0.046624	321.461	280.30	124.683
22.270	0.5658	0.046624	321.461	309.40	137.627
24.876	0.6318	0.046624	321.461	309.40	137.627
24.876	0.6318	0.046624	321.461	1201.70	534.540
27.176	0.6903	0.046624	321.461	1201.70	534.540
27.176	0.6903	0.046624	321.461	504.40	224.367
29.776	0.7563	0.046624	321.461	504.40	224.367
29.776	0.7563	0.046624	321.461	613.60	272.942
31.776	0.8071	0.046624	321.461	613.60	272.942
31.776	0.8071	0.046624	321.461	699.90	311.329
33.776	0.8579	0.046624	321.461	699.90	311.329
33.776	0.8579	0.046624	321.461	2350.40	1045.504
34.900	0.8864	0.046624	321.461	2350.40	1045.504
34.900	0.8864	0.071398	492.272	2350.40	1045.504
36.576	0.9290	0.071398	492.272	2350.40	1045.504
36.576	0.9290	0.071398	492.272	841.88	374.485
39.976	1.0153	0.071398	492.272	841.88	374.485
39.976	1.0153	0.071398	492.272	689.30	306.614
41.176	1.0458	0.071398	492.272	689.30	306.614
41.176	1.0458	0.071398	492.272	316.70	104.874
41.576	1.0560	0.071398	492.272	316.70	104.874
41.576	1.0560	0.071398	492.272	576.40	256.394
44.276	1.1245	0.071398	492.272	576.40	256.394
44.276	1.1245	0.071398	492.272	316.70	104.874
45.400	1.1531	0.071398	492.272	316.70	104.874
45.400	1.1531	0.049883	343.931	316.70	104.874
48.176	1.2236	0.049883	343.931	316.70	104.874
48.176	1.2236	0.049883	343.931	576.40	256.394
53.176	1.3506	0.049883	343.931	576.40	256.394
53.176	1.3506	0.049883	343.931	316.70	104.874
116.576	2.9610	0.049883	343.931	316.70	104.874
116.576	2.9610	0.049883	343.931	576.40	256.394
118.000	2.9971	0.049883	343.931	576.40	256.394
118.000	2.9971	0.059360	410.486	576.40	256.394
119.576	3.0372	0.059360	410.486	576.40	256.394
119.576	3.0372	0.059360	410.486	316.70	104.874
177.076	4.4977	0.059360	410.486	316.70	104.874

TABLE I (Continued)
PHYSICAL CHARACTERISTICS OF LAUNCH VEHICLE

x, in.	x, meter	$m,$ $\frac{lb \cdot sec^2}{in^2}$	$m,$ $\frac{Newton \cdot sec^2}{meter^2}$	AE 1b $\times 10^{-6}$	AE Newton $\times 10^{-1}$
177.076	4.4977	0.059360	410.486	576.40	256.394
180.076	4.5739	0.059360	410.486	576.40	256.394
180.076	4.5739	0.059360	410.486	316.70	140.874
185.176	4.7034	0.059360	410.486	316.70	140.874
185.176	4.7034	0.059360	410.486	841.90	374.494
187.676	4.7669	0.059360	410.486	841.90	374.494
187.676	4.7669	0.059360	410.486	316.70	140.874
187.776	4.7694	0.059360	410.486	316.70	140.874
187.776	4.7694	0.059360	410.486	1041.30	463.191
188.776	4.7948	0.059360	410.486	1041.30	463.191
188.776	4.7948	0.059360	410.486	2449.20	1089.453
189.100	4.8031	0.059360	410.486	2449.20	1089.453
189.100	4.8031	0.100380	692.096	2449.20	1089.453
192.776	4.8964	0.100380	692.096	2449.20	1089.453
192.776	4.4951	0.100380	692.096	39.00	17.348
193.100	4.9047	0.100380	692.096	39.00	17.348
193.100	4.9047	0.007668	52.869	39.00	17.348
201.156	5.1093	0.007668	52.869	39.00	17.348
201.156	5.1093	0.007668	52.869	78.00	34.696
282.200	5.1358	0.007668	52.869	78.00	34.696
282.200	5.1358	0.018549	127.891	78.00	34.696
203.756	5.1753	0.018549	127.891	78.00	34.696
203.756	5.1753	0.018549	127.891	112.80	50.176
204.200	5.1866	0.018549	127.891	112.80	50.176
204.200	5.1866	0.012378	85.343	112.80	50.176
209.382	5.3182	0.012378	85.343	112.80	50.176
209.382	5.3182	0.012378	85.343	135.20	60.140
211.700	5.3771	0.012378	85.343	135.20	60.140
211.700	5.3771	0.025331	174.651	135.20	60.140
214.300	5.4432	0.025331	174.651	135.20	60.140
214.300	5.4432	0.025331	174.651	369.20	164.228
215.100	5.4635	0.025331	174.651	369.20	164.228
215.100	5.4635	0.025331	174.651	130.00	57.827
219.556	5.5767	0.025331	174.651	130.00	57.827
219.556	5.5767	0.025331	174.651	118.60	52.756
221.716	5.6333	0.025331	174.651	118.60	52.756
221.716	5.6333	0.025331	174.651	169.80	75.174
223.200	5.6692	0.025331	174.651	169.80	75.174
223.200	5.6692	0.023840	164.330	169.80	75.174
224.500	5.7022	0.023840	164.330	169.80	75.174
224.500	5.7022	0.023840	164.330	442.00	196.610
225.356	5.7230	0.023840	164.330	442.00	196.610
225.356	5.7230	0.023840	164.330	280.00	124.905

TABLE I (Continued)
PHYSICAL CHARACTERISTICS OF LAUNCH VEHICLE

x, in.	x, meter	m, lb-sec ² / in ²	m, Newton-sec ² meter ²	AE lb x10 ⁻⁶	AE Newton x10 ⁻¹
228.056	5.7926	0.023840	164.330	280.00	124.905
228.056	5.7926	0.023840	164.330	730.10	324.763
229.456	5.8281	0.023840	164.330	730.10	324.763
229.456	5.8281	0.023840	164.330	137.80	61.296
230.700	5.8597	0.023840	164.330	137.80	61.296
230.700	5.8597	0.024559	169.328	137.80	61.296
231.356	5.8764	0.024559	169.328	137.80	61.296
231.356	5.8764	0.024559	169.328	70.50	31.360
331.356	8.4164	0.024559	169.328	70.50	31.360
331.356	8.4164	0.024559	169.328	130.00	57.827
332.900	8.4556	0.024559	169.328	130.00	57.827
332.900	8.4556	0.044689	308.120	431.00	191.984
335.956	8.5332	0.044689	308.120	431.60	191.984
335.956	8.5332	0.044689	308.120	213.20	94.836
337.900	8.5840	0.044689	308.120	213.20	94.836
337.900	8.5840	0.013808	95.203	71.42	31.769
340.500	8.6486	0.013808	95.203	98.60	43.877
340.500	8.6486	0.013808	95.203	98.60	43.877
347.200	8.8188	0.013808	95.203	132.64	58.998
347.200	8.8188	0.025466	175.528	132.64	58.998
348.000	8.8391	0.025466	175.528	136.83	60.865
348.000	8.8391	0.025466	175.528	136.83	60.865
351.000	8.9153	0.025466	175.582	425.60	189.315
351.000	8.9153	0.025466	175.582	425.60	189.315
351.800	8.9357	0.025466	175.582	136.83	60.865
351.800	8.9357	0.025466	175.582	136.83	60.865
357.800	9.0881	0.025466	175.582	112.60	50.087
357.800	9.0881	0.025466	175.582	112.60	50.087
361.200	9.1744	0.025466	175.582	199.10	88.564
361.200	9.1744	0.022530	155.339	612.00	272.230
362.300	9.2024	0.022530	155.339	199.10	88.564
362.300	9.2024	0.022530	155.339	199.10	88.564
364.800	9.2659	0.022530	155.339	336.70	149.771
364.800	9.2659	0.022530	155.339	730.10	324.763
366.200	9.3014	0.022530	155.339	730.10	324.763
366.200	9.3014	0.024559	169.328	195.00	86.740
368.100	9.3497	0.024559	169.328	70.50	31.360
368.100	9.3497	0.024559	169.328	70.50	31.360
467.700	11.8795	0.024559	169.328	70.50	31.360
467.700	11.8795	0.044689	308.120	70.50	31.360
468.100	11.8897	0.044689	308.120	70.50	31.360
468.100	11.8897	0.044689	308.120	70.50	31.360
469.700	11.9303	0.044689	308.120	184.50	82.069

TABLE I (Continued)
PHYSICAL CHARACTERISTICS OF LAUNCH VEHICLE

x, in.	x, meter	m, lb-sec ² / in ²	m, Newton-sec ² / meter ²	AE lb x10 ⁻⁶	AE Newton x10 ⁻¹
469.700	11.9303	0.044689	308.120	184.50	82.069
471.800	11.9837	0.044689	308.120	529.10	235.354
471.800	11.9837	0.004788	308.120	529.10	235.354
472.700	12.0065	0.004788	308.120	529.10	235.354
472.700	12.0065	0.004788	308.120	187.80	83.537
474.600	12.0548	0.004788	308.120	187.80	83.537
474.600	12.0548	0.004788	308.120	75.40	33.539
477.200	12.1208	0.004788	308.120	75.40	33.539
477.200	12.1208	0.004788	308.120	707.20	314.577
477.500	12.1285	0.004788	308.120	707.20	314.577
477.500	12.1285	0.004788	308.120	75.40	33.539
481.400	12.2275	0.004788	308.120	75.40	33.539
481.400	12.2275	0.004788	308.120	75.40	33.539
483.400	12.2783	0.004788	308.120	216.60	93.679
483.400	12.2783	0.004788	308.120	190.90	84.916
484.700	12.3113	0.004788	33.012	452.40	201.236
484.700	12.3113	0.023784	163.378	452.40	201.236
484.749	12.3380	0.023784	163.378	870.70	387.305
484.749	12.3380	0.023784	163.378	870.70	387.305
485.811	12.3396	0.023784	163.378	870.70	387.305
485.811	12.3396	0.028338	193.605	162.80	72.417
486.749	12.3634	0.028338	193.605	130.00	57.829
486.749	12.3634	0.034161	235.532	130.00	57.829
487.624	12.3856	0.034161	234.642	98.50	43.815
487.624	12.3856	0.017728	122.230	33.46	14.884
488.120	12.3982	0.017728	122.230	33.46	14.884
488.120	12.3982	0.017728	122.230	33.46	14.884
490.749	12.4650	0.017728	122.230	33.46	14.884
490.749	12.4650	0.012320	77.442	33.46	14.884
491.999	12.4967	0.012320	76.376	33.46	14.884
491.999	12.4967	0.008230	73.540	33.46	14.884
494.085	12.6513	0.008230	56.774	33.46	14.884
494.085	12.5513	0.014880	102.594	33.46	14.884
498.549	12.6631	0.014880	102.594	33.46	14.884
498.549	12.6631	0.011827	87.432	33.46	14.884
504.085	12.8037	0.011827	81.544	33.46	14.884
504.085	12.8037	0.005952	41.038	33.46	14.884
504.585	12.8164	0.018245	43.540	33.46	14.884
504.585	12.8164	0.018245	125.795	33.46	14.884
505.499	12.8396	0.018245	125.795	33.46	14.884
505.499	12.8396	0.018245	125.795	182.16	81.028
506.599	12.8676	0.018245	125.795	182.16	81.028
506.835	12.8736	0.045031	310.478	25.74	11.450
507.800	12.8981	0.045031	310.478	25.74	11.450

TABLE I (Continued)
PHYSICAL CHARACTERISTICS OF LAUNCH VEHICLE

x, in.	x, meter	m, lb-sec ² / in ²	m, Newton-sec ² meter ²	AE 1b x10 ⁻⁶	AE Newton x10 ⁻¹
507.800	12.8981	0.045031	310.478	4.45	1.979
511.899	13.0022	0.045031	310.478	4.20	1.294
511.899	13.0022	0.051756	356.845	4.20	1.294
513.099	13.0372	0.051756	356.845	4.15	1.673
513.099	13.0372	0.045316	312.443	4.15	1.673
536.589	13.6293	0.045316	312.443	3.40	1.512
536.589	13.6293	0.046558	321.006	3.40	1.512
537.909	13.6651	0.046558	321.006	3.35	1.490
537.909	13.6651	0.004710	32.740	3.35	1.490
542.589	13.7817	0.004710	32.740	3.20	1.423
542.589	13.7817	0.015931	109.846	3.20	1.423
546.899	13.8912	0.015931	109.846	3.15	1.401
546.899	13.8912	0.027685	190.881	3.15	1.401
584.849	13.9407	0.027685	190.991	3.10	1.378
584.849	13.9407	0.005435	37.473	3.10	1.378
553.069	14.0479	0.005435	37.473	2.91	1.294
553.069	14.0479	0.005435	40.148	2.91	1.294
554.609	14.0870	0.005435	40.148	2.71	1.205
554.609	14.0870	0.006185	42.644	2.71	1.205
555.999	14.1223	0.005900	40.679	2.41	1.071
555.999	14.1223	0.005900	40.679	2.41	1.071
559.068	14.2003	0.005176	35.687	1.91	0.849
559.068	14.2003	0.005176	35.687	1.91	0.849
560.999	14.2493	0.005176	35.687	1.61	0.716
560.999	14.2493	0.005176	35.687	1.61	0.716
564.109	14.3283	0.004790	33.026	1.42	0.631
564.109	14.3283	0.008851	61.025	1.42	0.631
565.068	14.3527	0.008800	60.674	1.37	0.609
565.068	14.3527	0.008800	60.674	1.37	0.609
567.800	14.4221	0.008800	60.674	1.00	0.448
567.800	14.4221	0.008800	60.674	1.00	0.448
569.427	14.4634	0.008644	59.598	0.782	0.347
569.427	14.4634	0.008644	59.598	0.782	0.347
569.857	14.4744	0.001941	13.383	0.60	0.266
569.857	14.4744	0.001941	13.383	0.60	0.266
575.749	14.6240	0.001579	10.887	0.398	0.177
575.749	14.6240	0.000924	6.371	0.398	0.177
577.800	14.6761	0.000700	4.826	0.36	0.160
577.800	14.6761	0.000700	4.826	0.26	0.115
580.999	14.7573	0.000655	4.516	0.210	0.0934
580.999	14.7573	0.000655	4.516	0.210	0.0934
581.800	14.7781	0.000500	3.447	0.200	0.0098
581.800	14.7781	0.000500	3.447	0.200	0.0088

TABLE I (Concluded)
PHYSICAL CHARACTERISTICS OF LAUNCH VEHICLE

x, in.	x, meter	$\frac{m}{lb \cdot sec^2}$ / in^2	$\frac{m}{Newton \cdot sec^2}$ / $meter^2$	AE 1b $\times 10^{-6}$	AE N $\times 10^{-1}$
587.420	14.9204	0.000466	3.447	0.169	0.0751
587.420	14.9204	0.000466	3.447	0.161	0.0716
589.260	14.9672	0.001633	10.811	2.000	0.8896
589.260	14.9672	0.001633	10.811	2.000	0.8896
590.127	14.9892	0.001633	10.811	4.620	2.0549
590.127	14.9892	0.001633	10.811	4.620	2.0549
593.429	15.0713	0.001633	0.000	0.100	0.0444

TABLE II
PHYSICAL CHARACTERISTICS OF PAYLOAD

(a) U.S. Customary Units

$x,$ in	$m,$ $lb \cdot sec^2/in^2$	$AE \times 10^{-6}$ lb
488.320	0.02	2.0000
548.120	0.02	2.0000

(b) SI Units

$x,$ meter	$m,$ $Newton \cdot sec^2$ meter ²	$AE \times 10^{-7}$ Newton
12.3983	137.8951	0.8896
13.9223	137.8951	0.8896

TABLE III

A COMPARISON OF THE NATURAL LONGITUDINAL FREQUENCIES CALCULATED BY THE LUMPED MASS METHOD OF REFERENCE 1 AND THE FINITE DIFFERENCE PROCEDURE FOR THE ROCKET-VEHICLE CONFIGURATION

Mode	Lumped Mass (Hz)	Finite Difference (Hz)	Percent Increase
1	36.6	38.5	5.2
2	47.7	49.7	4.2
3	72.6	75.0	4.6

NORMALIZED DISPLACEMENT

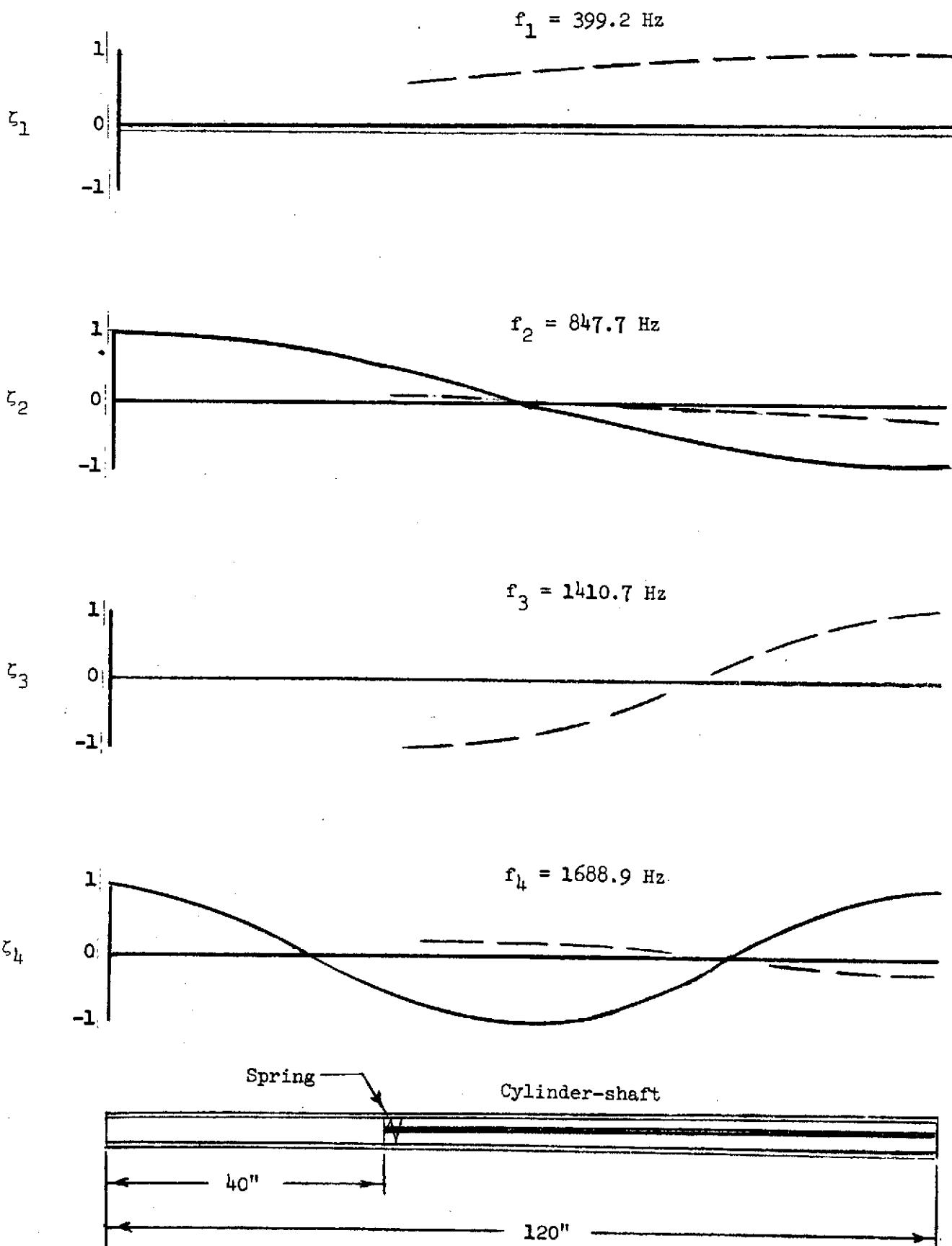


Figure 1.- Numerical example natural longitudinal mode shapes.

NORMALIZED DISPLACEMENT

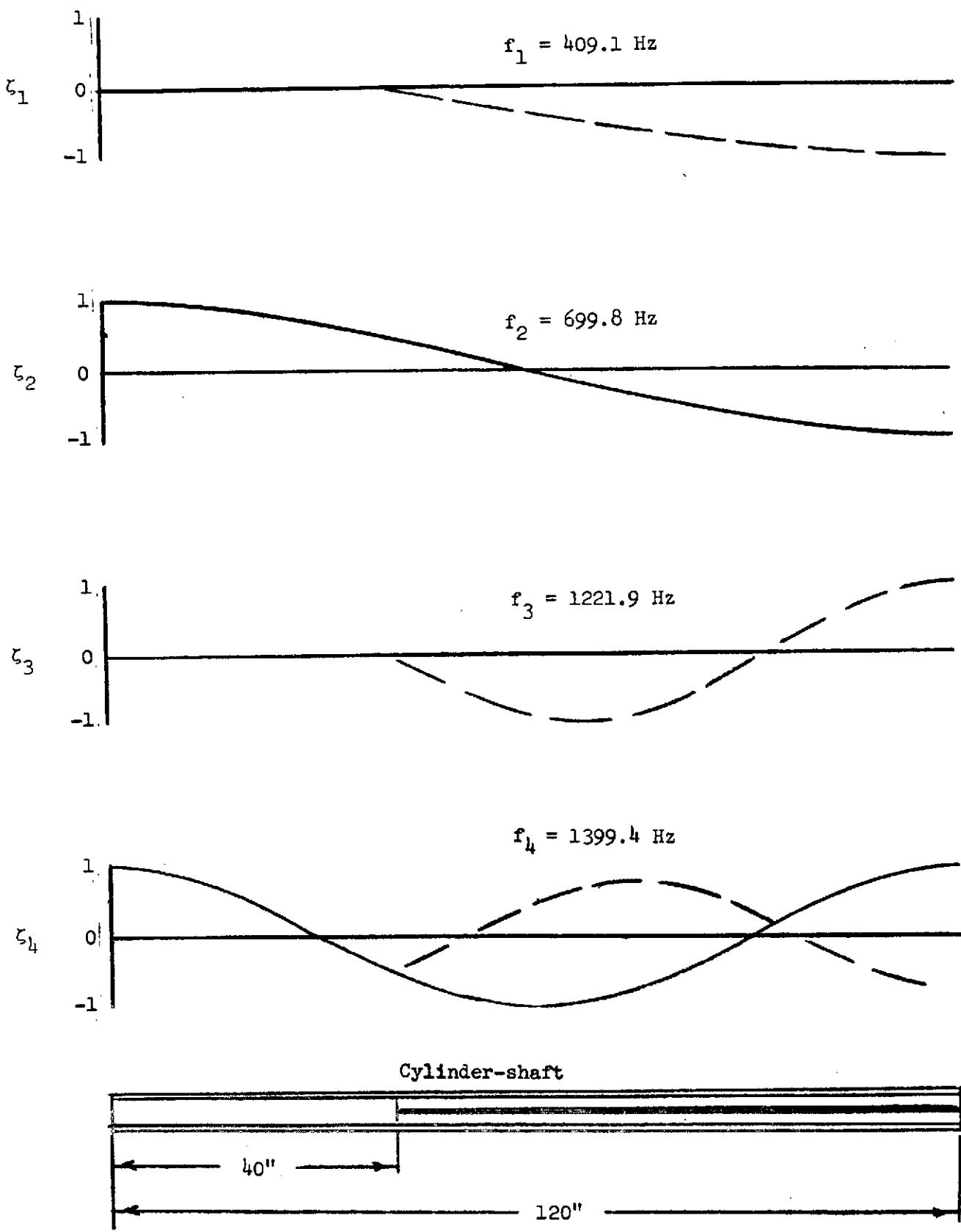


Figure 2 .- Numerical example natural torsional mode shapes.

E7

CALCOMP PLOTTING INSTRUCTION CARD

NAME <i>Maria Stegman</i>		DIV. SEA	BLDG. NO. 586	PHONE 2512
MAIL STOP <i>311</i>		BIN NO.	J.O. NO. <i>R2959</i> DATE <i>8-30-73</i>	
NO. OF PLOTS <i>9</i>	PLOT STOP ADDRESS	<i>999</i>	PLOT MODE	<input type="checkbox"/> Single PEN <input type="checkbox"/> Ballpoint <input checked="" type="checkbox"/> Multiple TYPE <input type="checkbox"/> Leroy, Size _____
INK COLOR	Black	Blue	Red	Green
				PAPER NO. <i>400</i>
				PAPER TYPE <i>Rastbond</i>
STARTING LOCATION	X Inches <i>1.0</i>	SPECIAL INSTRUCTIONS		
	Y Inches <i>1.0</i>	<i>Return to B586, R-209 TUB</i>		
PROCESS NO. <i>6810</i>	CAL 303 TAPE NO. <i>R2959</i>	2219 OFF ON	NASA Langley (Feb. 1970) ACD-OCC	

Figure 3.- Calcomp plotting instruction card.

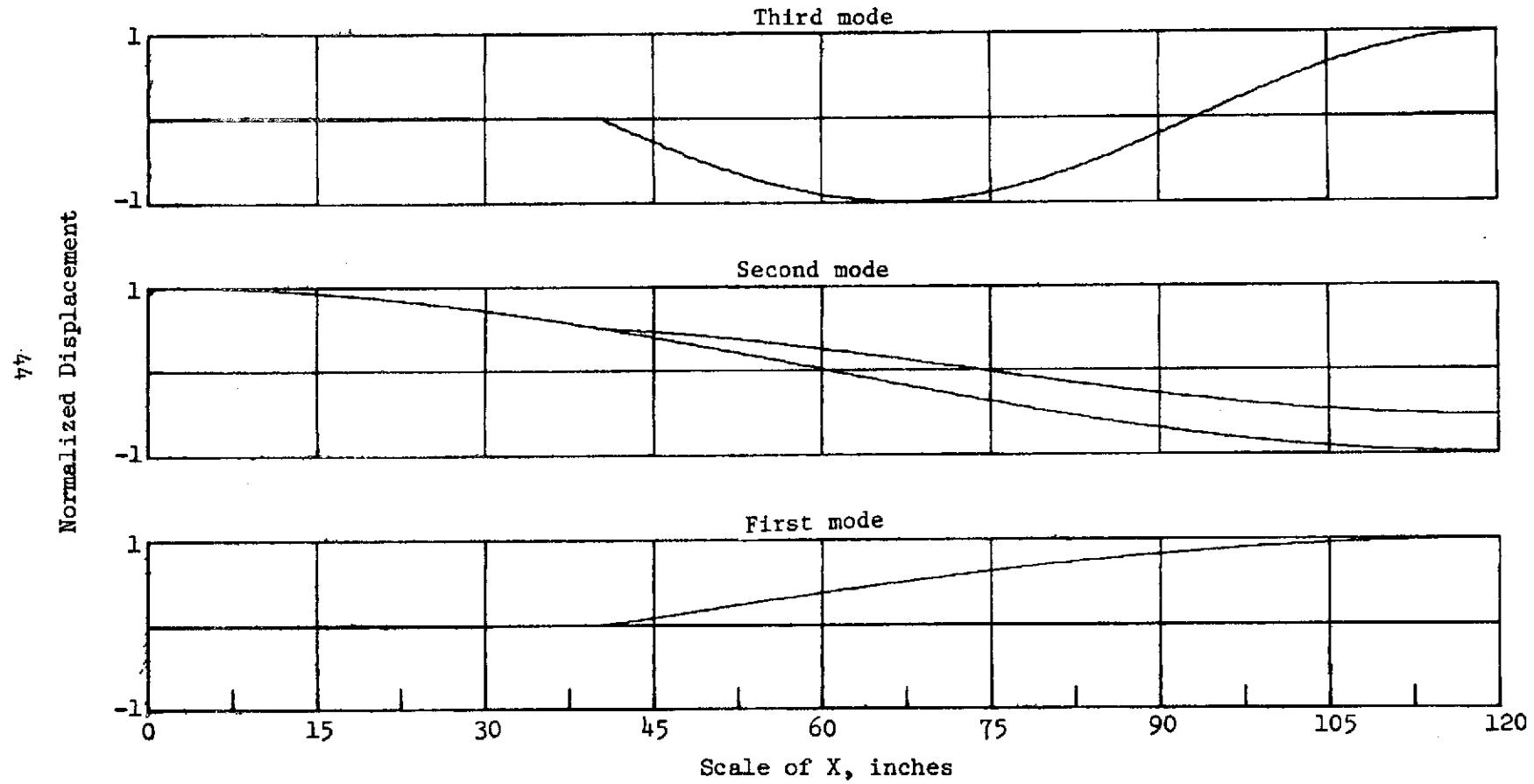


Figure 4.- Numerical example plot of torsional mode shapes.

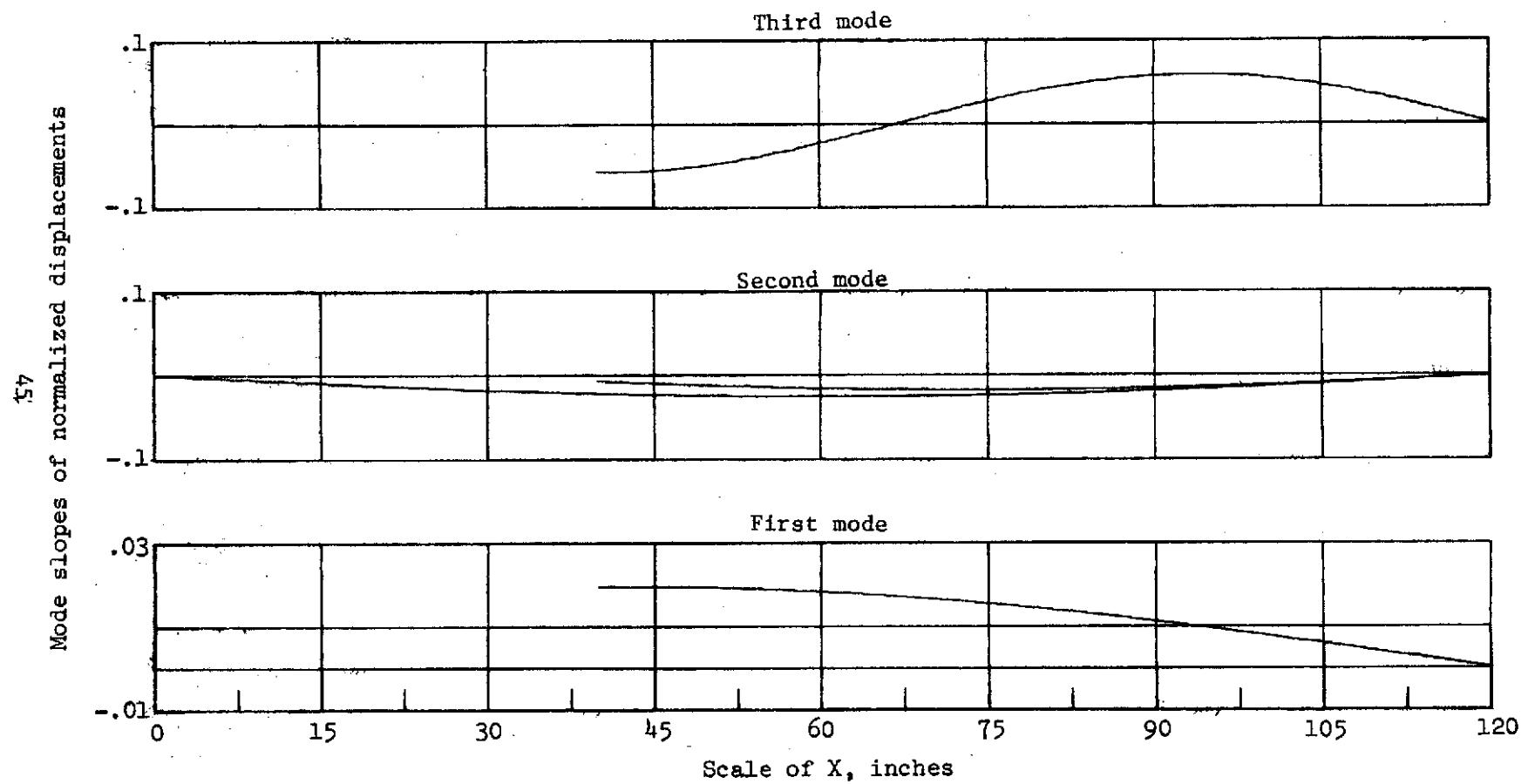


Figure 5.- Numerical example plot of torsional mode slopes.

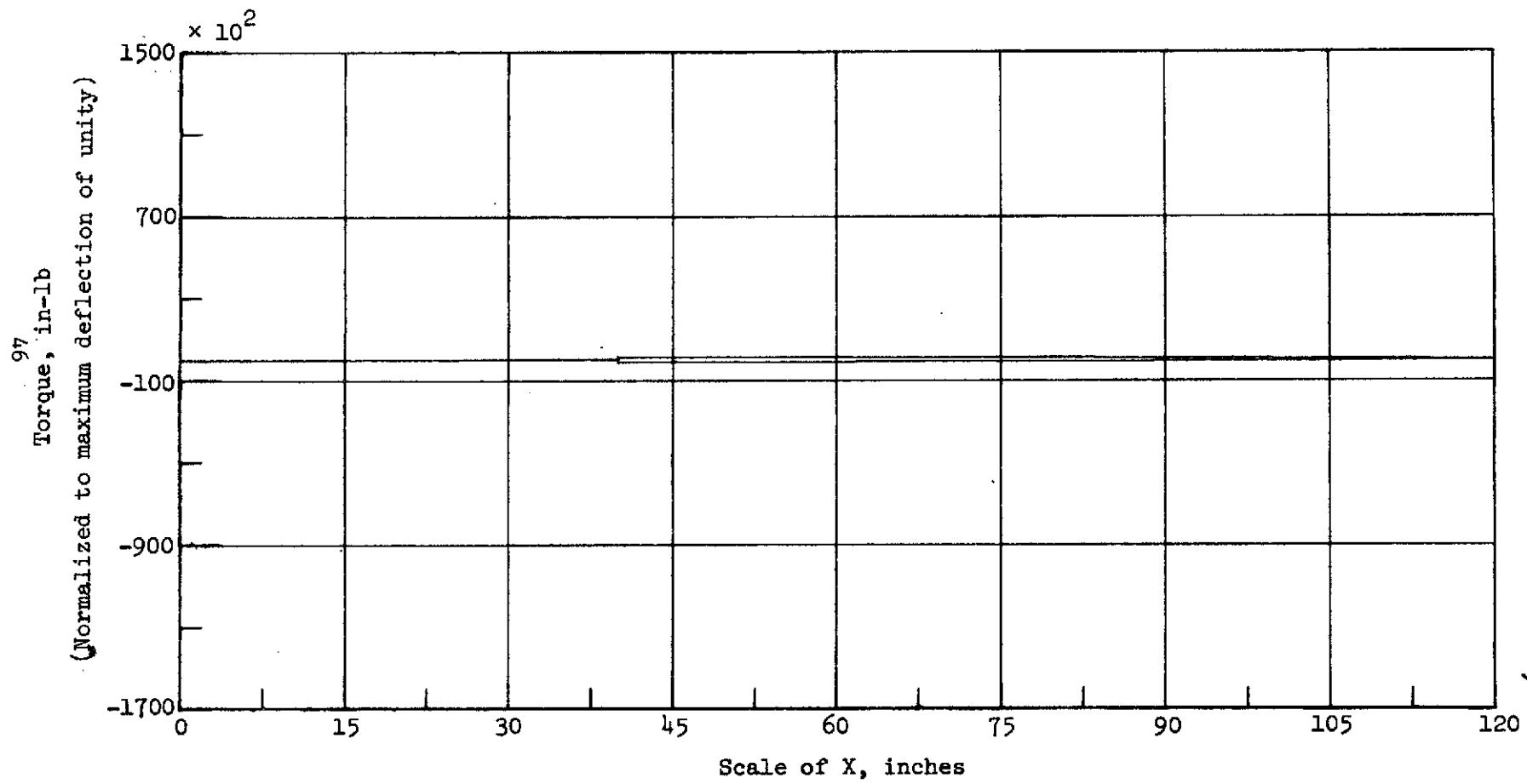


Figure 6.- Numerical example plot of torque distribution for first mode.

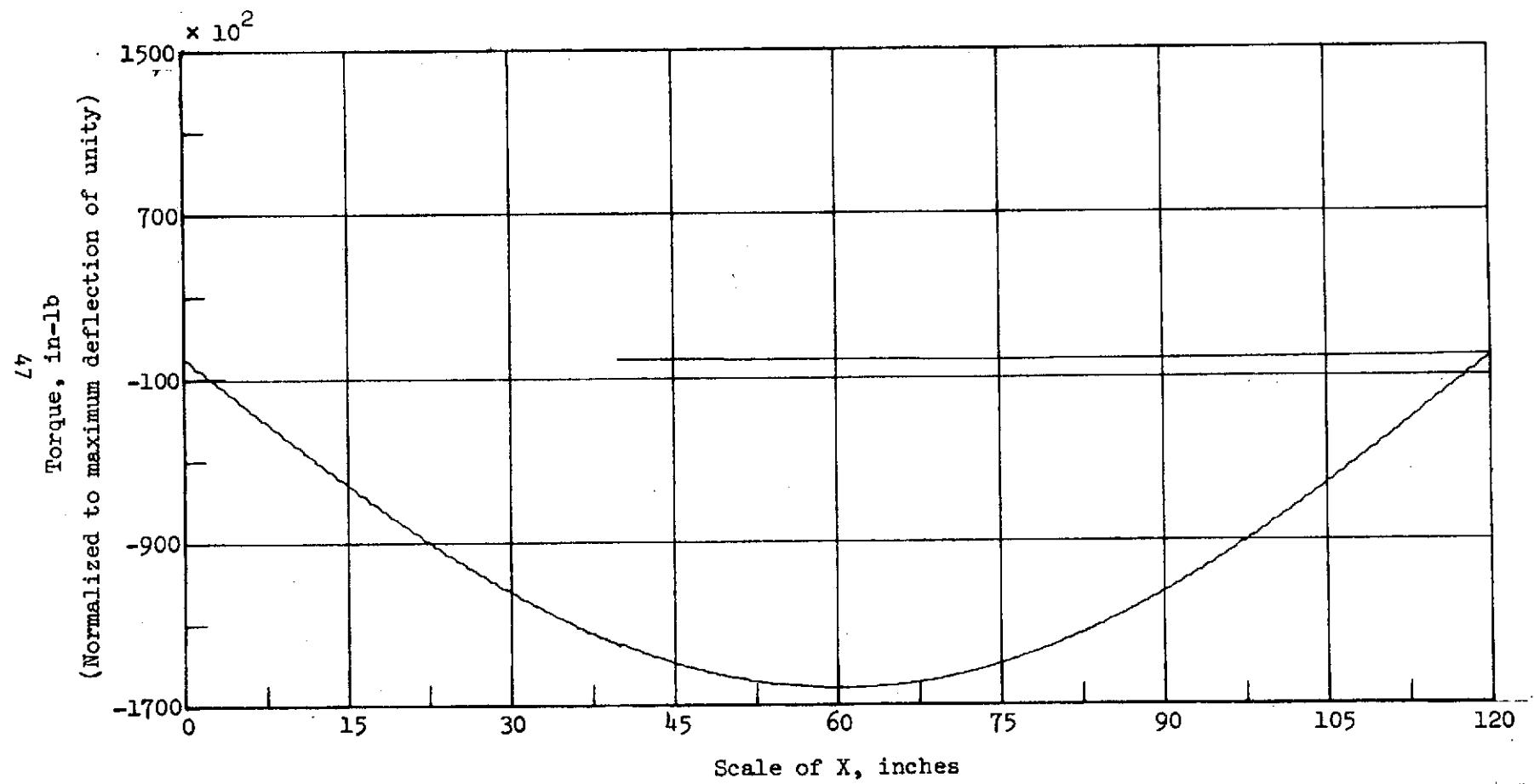


Figure 7 .- Numerical example plot of torque distribution for second mode.

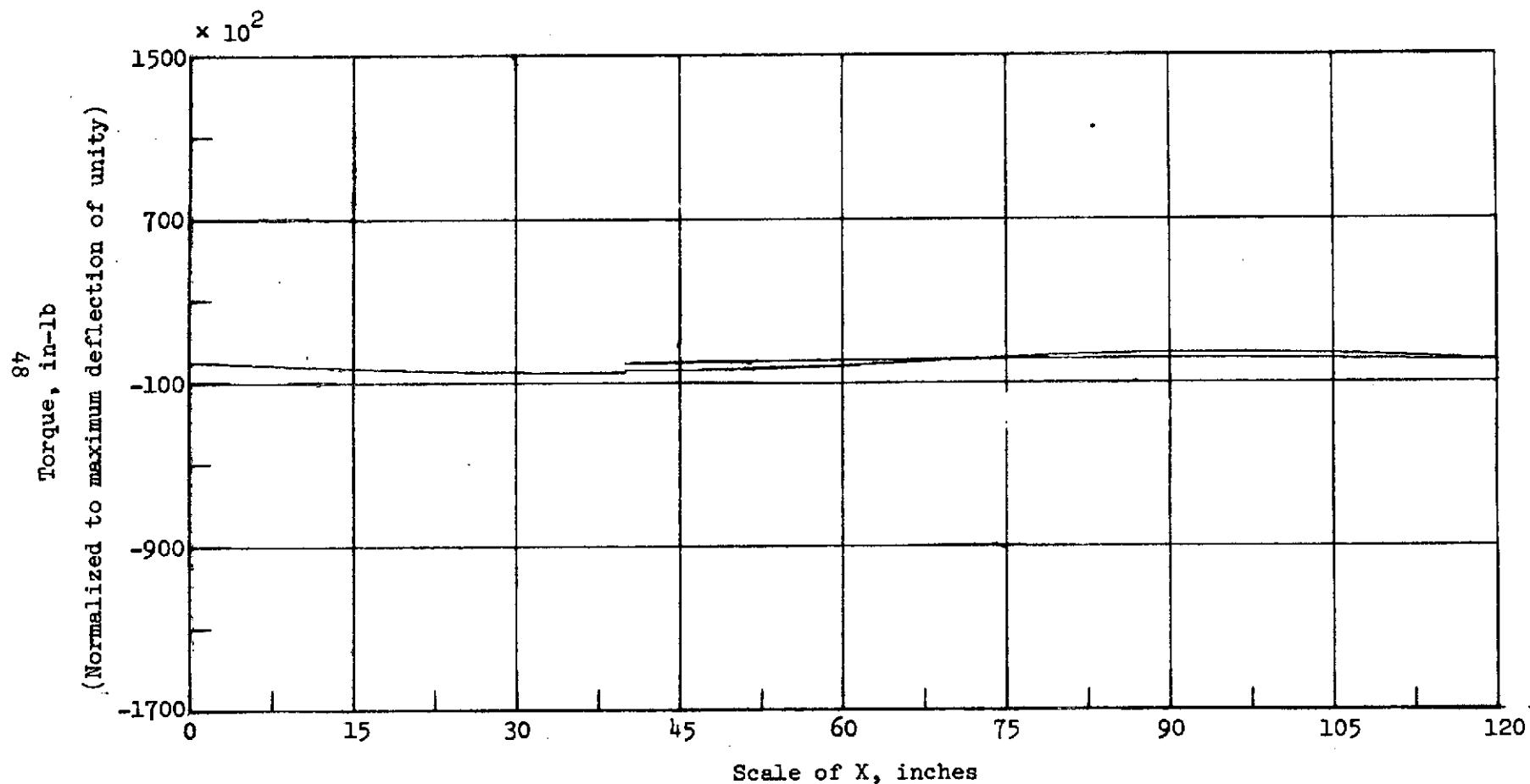


Figure 8.- Numerical example plot of torque distribution for third mode.

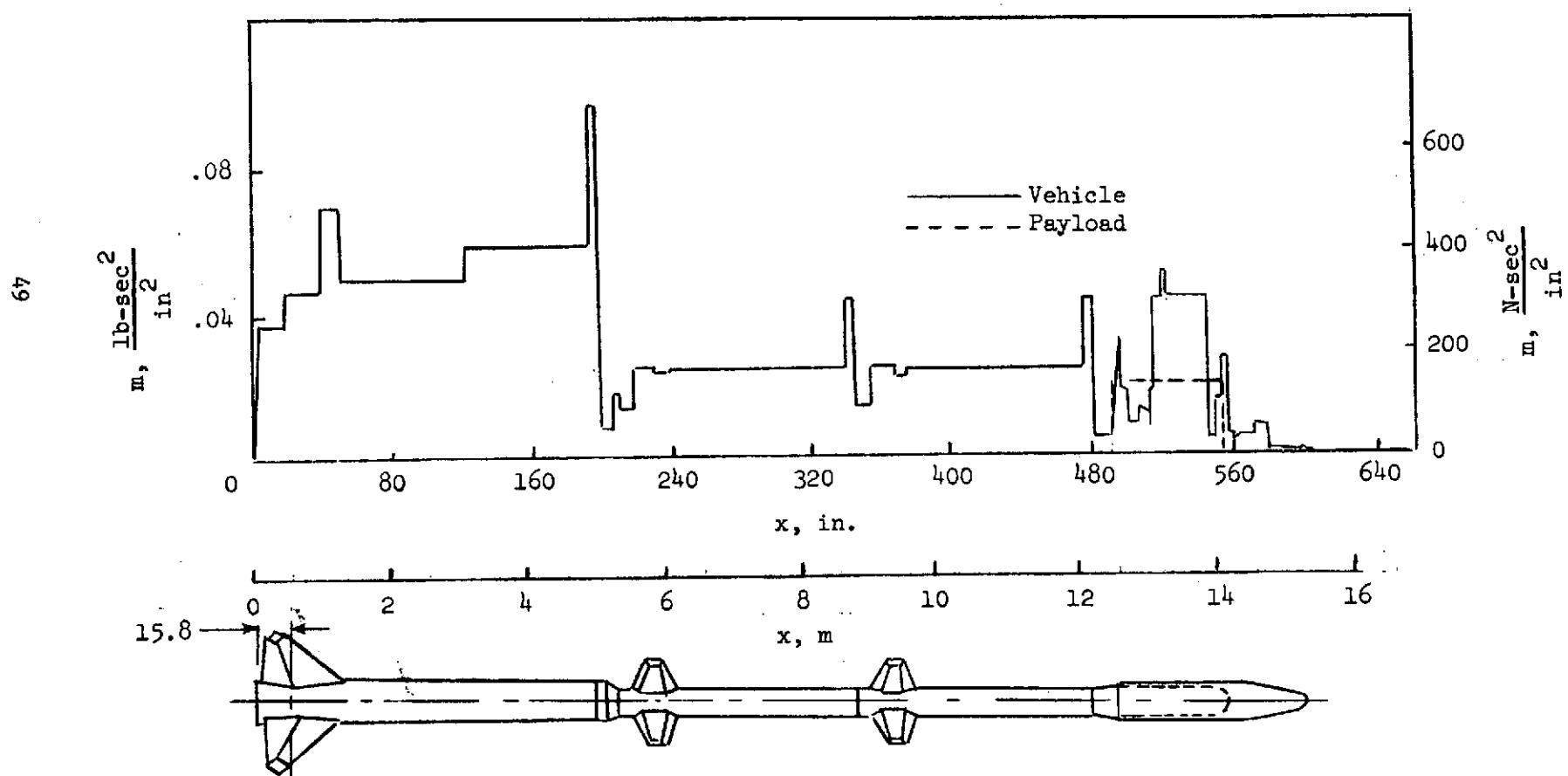


Figure 9.- Rocket-vehicle mass per inch. m values from tables I and II.

05

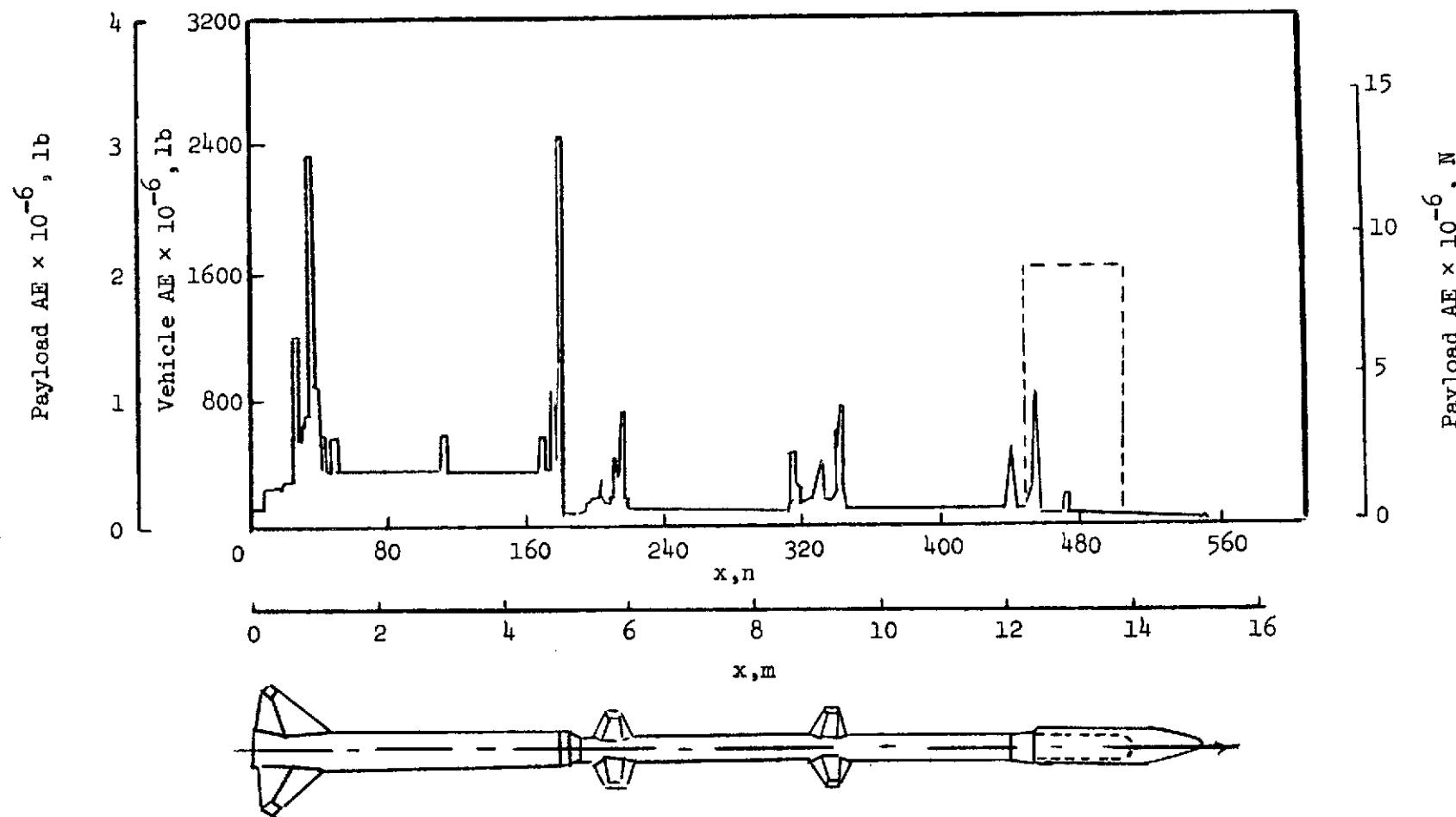


Figure 10.- Rocket-vehicle axial extension coefficient. AE values from tables I and II.

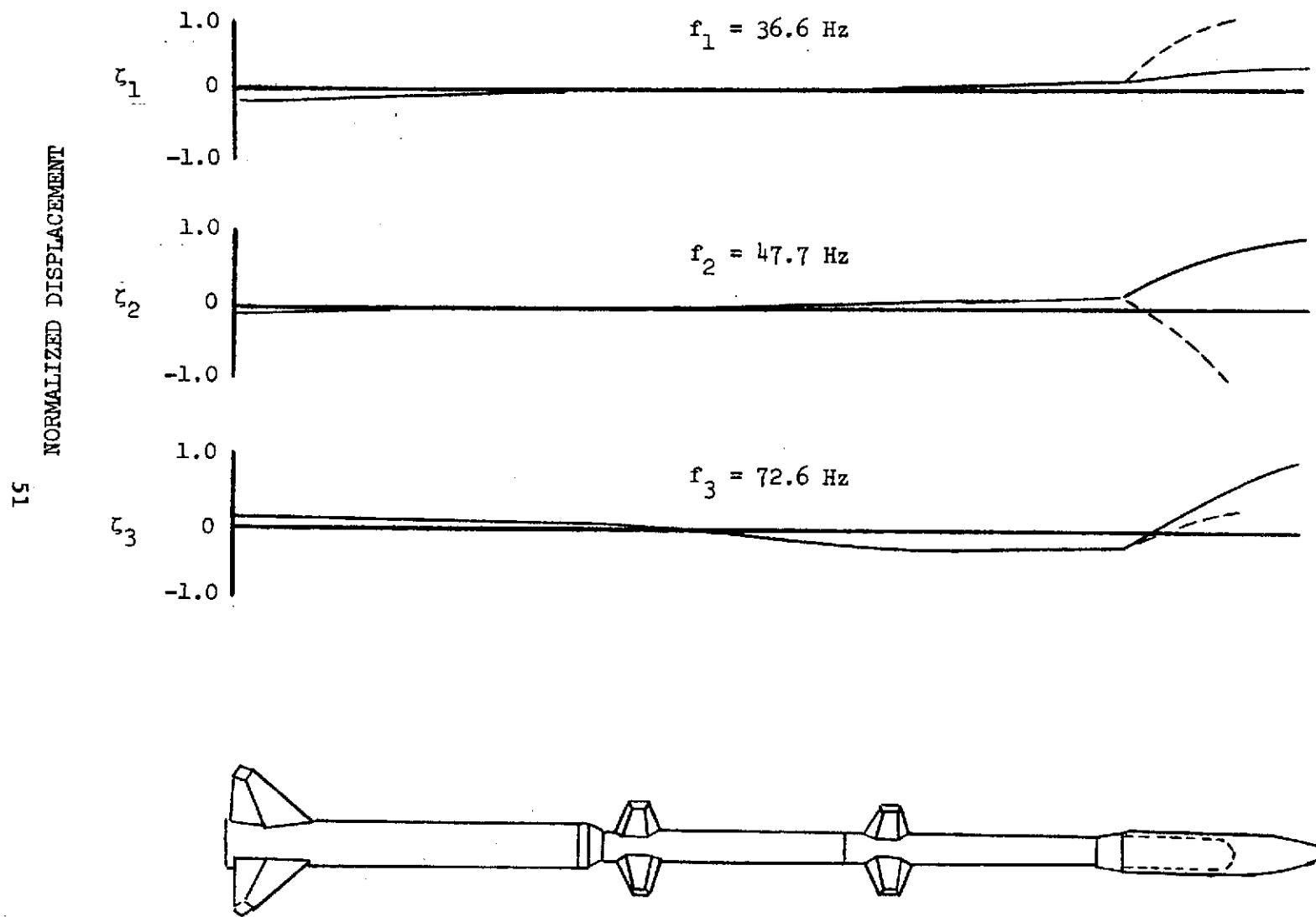


Figure 11.- Rocket-vehicle longitudinal natural mode shapes.

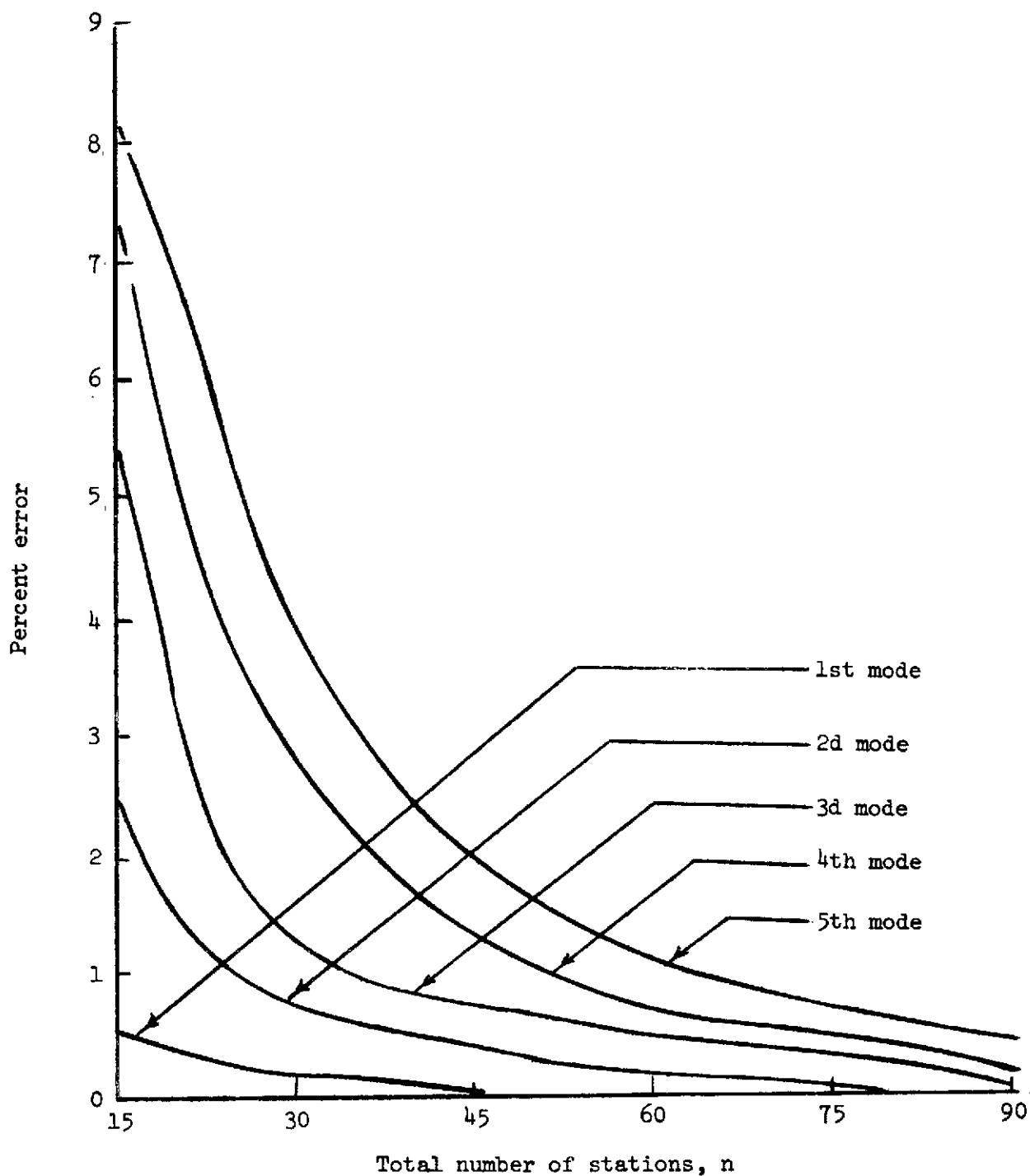
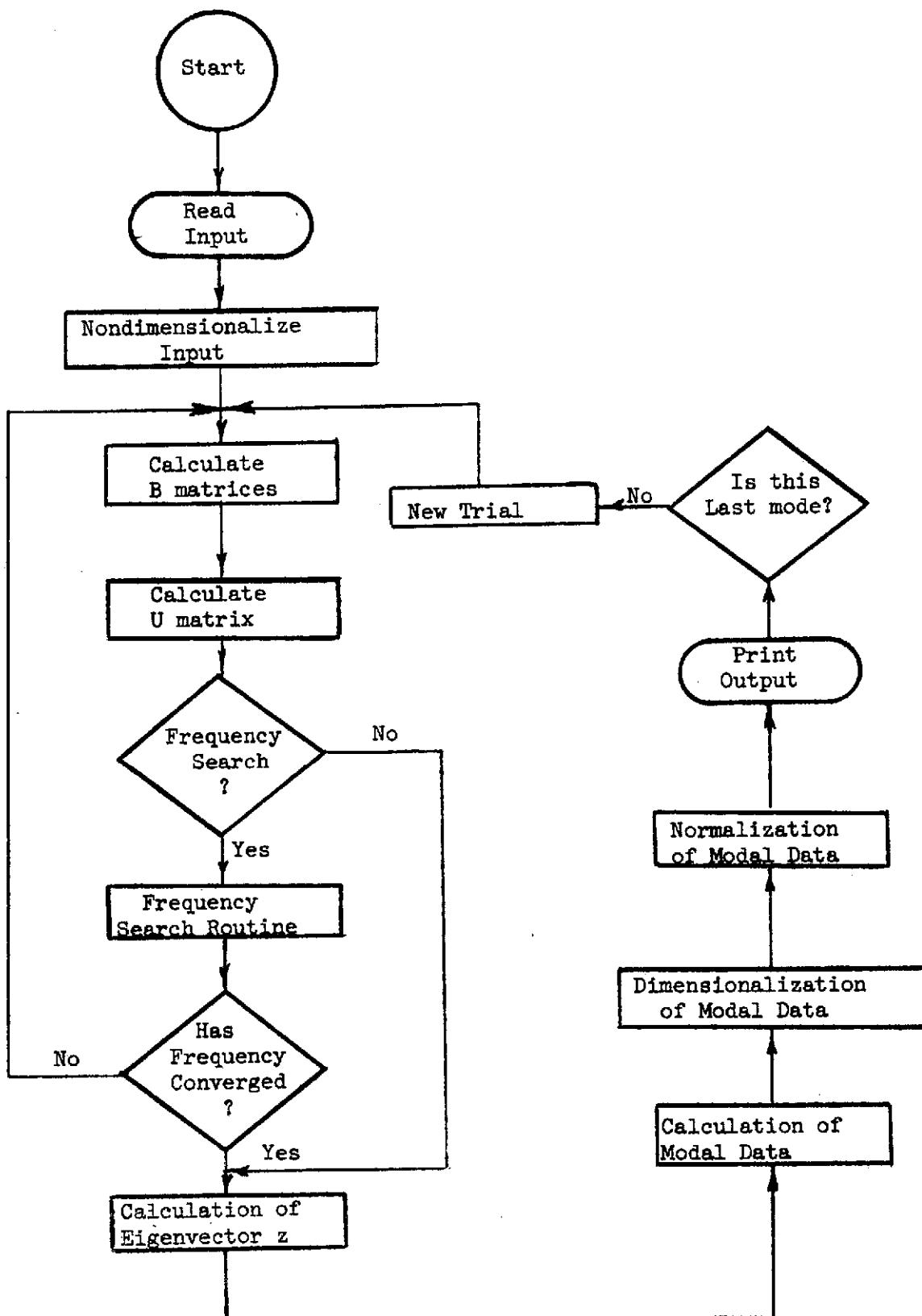


Figure 12.- Frequency percent error as a total number of stations.



Modular Flow Chart

Figure 13.- Computer program flow diagram.

APPENDIX A

DESCRIPTION OF COMPUTER PROGRAM

METHOD OF INPUT

1. One or two title cards are allowed. Columns 1 to 80 may be used on each card. Both title cards will be printed at the beginning of output. Thereafter only the first title card will be used as a header.
2. The following namelists may be read in any order.

a. Namelist name: \$LONVIB

Variable list: X, MASS, AE, NON, TIME, BEAM

Remarks: Each beam member is assigned a number (by the program) which depends upon the order in which the \$LONVIB namelist groups are input. \$LONVIB for beam member #3 must follow \$LONVIB for beam member #2, for instance.

Note: Beam is never used in the program, so need not be input. The last value read in for TIME will be printed in the output for each mode. This name list is applicable when longitudinal vibration analysis is desired.

b. Namelist name: \$TORVIB

Variable list: X, ZR, JG, NON, TIME, BEAM

Remarks: Each beam member is assigned a number (by the program) in the same manner as described for \$LONVIB. This namelist is applicable when torsional vibration analysis is desired.

c. Namelist name: \$INPFRQ

Variable list: OMEG

Remarks: This namelist group is input only if the natural frequencies are known. No frequency search will be conducted when this group is input.

d. Namelist name: \$MODES

Variable list: OMEGA, DELOMG, OMGTOL, NOMODE, FRSTMD, NORM, NORMBR, NORDIS, NOREQM

Remarks: In every run, this group must be entered once. If entered when \$INPFRQ is entered, the first four variables in this namelist group will be ignored.

Note that OMGTOL is a relative difference in frequencies

computed by this formula, $\text{OMGTOL} = \left| \frac{\omega_1}{\omega_2} - 1 \right|$.

Note that NOMODE is not affected by the value of FRSTMD. FRSTMD is set to one (1) internally, so need be entered only when the set of numbers assigned to the modes should not begin with unity; for example when a run is to start with other than the first mode.

To normalize to a particular station on one beam member, set NORM equal to that station number and set NORMBR equal to that beam member number. To normalize to the station of maximum modal displacement, omit NORM and NORMBR and set NORDIS equal to "T". To normalize to the generalized mass, omit NORM, NORMBR and NORDIS and set NOREQM equal to "T". To obtain non-normalized data, omit NORM, NORMBR, NORDIS, and NOREQM.

To obtain a printed list of the nondimensionalized frequency and $|U|$ values calculated during the frequency search, insert a minus sign in front of the value for NOMODE.

e. Namelist name: \$JOINT

Variable list: MBRI(J), MBRJ(J), MBRK(J), MBRL(J), IOJTRN(J), TRNS(J), IJTRN(J), IKTRN(J), ILTRN(J), TRNFIJ(J), TRNFIK(J), TRNFIL(J).

Remarks: This group must be input once for each run. It describes which members are constrained at each joint. If the constraint is partial, the flexibility of the partial constraint must be entered. MBRI(J), MBRJ(J), MBRK(J), MBRL(J), and IOJTRN(J) must always be entered.

The constraint codes are:

1 = Full constraint
2 = Partial constraint
3 = No constraint

f. Namelist name: \$CTRL

Variable list: RFX, RFMASS, RFAE, XMOD, MASMOD, AEMOD, DELX, JGMOD, ZRMOD, RFJG, RFZR.

Remarks: This namelist group controls the accuracy of the solution. Only those variables not equal to one must be entered, since these variables are initially set equal to one by the program.

g. Namelist name: \$OUTPUT

Variable list: ZTAC, DZTAC, TENS, DTENS, TORQ, DTORQ, ISID, ISOD, PLOT.

Remarks: This group must be input once in each run. Four of the first six variables listed above must be input. ISID, ISOD, and PLOT are set to F internally so need be entered only if T. If a Calcomp plot is desired, PLOT = T, and items 3 and 4 of INPUT must follow the \$END card.

h. Namelist name: \$END

Variable list: None.

Remarks: Last card input of section 2 of INPUT must contain \$END. If PLOT = T, in input namelist g of INPUT, then items 3 and 4 must follow.

3. Header card for Calcomp plot identification. Use columns 1 to 80 on card.

4. Namelist name: \$NAM1

Variable list: INCHX, DX, XMIN, PLTZ, PLTZPR, NZAPR, MPLT, PLTT, NPLTT, TMIN, ZPMIN1, DZP1, ZPMIN2, DZP2, ZPMIN3, DZP3, DTT, PLTM, NMZR, MZRMIN, MZRMIN, DMZR, PLTZR, PLTAE, PLTGJ, NMAEJG, AJMIN, DAEJG.

Remarks: The header card and \$NAM1 namelist must be read for every case separately if plots are desired.

SYMBOLS

Input Namelist Names

\$CTRL	<u>control.</u> Input of numerical control data.
\$END	<u>end.</u> Required last item of input.
\$INPFRQ	<u>input frequency.</u> Input of known modal frequencies.
\$JOINT	<u>joint.</u> Input of joint description data.
\$LONVIB	<u>longitudinal vibration.</u> Input data characteristics of mass and axial stiffness.
\$MODES	<u>modes.</u> Input of data affecting selection of modes and normalization.
\$NAM1	plot information, if plots are desired.
\$OUTPUT	<u>output.</u> Input of data controlling the output of program results.
\$TORVIB	<u>torsional vibration.</u> Input data characteristics of torsional stiffness and polar mass moment of inertia.

NOMENCLATURE

A. Input-Output Variable Names of Main Program

In the following list, fixed point variables (no decimal point allowed) are designated by "Integer". Floating point variables (decimal point required) are designated by "Decimal". The use of letters rather than numbers is designated by "Logical".

<u>Name</u>	<u>Type</u>	<u>Description</u>
AE(N)	Decimal	<u>AE</u> . Axial stiffness at the Nth station, lb/in ² .
AEMOD	Decimal	<u>AE</u> modification. Constant by which each value of AE(N) will be multiplied, unitless.
BEAM	Integer	<u>beam</u> . In input, identification number of beam assigned by Physical Characteristics Program, unitless. It indicates the order in which the beams were input to (and output from) this program.
DELOMG	Decimal	<u>delta omega</u> . Frequency interval used in the search for modal frequencies, rad./sec.
DELX	Decimal	<u>delta X</u> . Maximum length of integration interval along vehicle center line, in.
DTENS	Integer	<u>derivative of tension</u> column number. Identification number for the column in the output in which DTEN(N) will be printed, unitless.
DTORQ	Integer	<u>derivative of torque</u> column number. Identification number for the column in the output in which torque-prime will be printed, unitless.
DZETA(N)	Decimal	<u>derivative of zeta</u> . Lengthwise derivative of modal deflection at the Nth station, in./in. (before normalization).
DZTAC	Integer	<u>derivative of zeta</u> column. Identification number for the column in the output in which DZETA(N) will be printed, unitless.
FRSTMD	Integer	<u>first mode</u> . Number to be assigned to the first mode calculated, unitless.

<u>Name</u>	<u>Type</u>	<u>Description</u>
IJTRN(J)	Integer	<u>i - j</u> <u>translation</u> . Code number for the translational constraint between beam members in the "i" and "j" positions at the J th joint, unitless. The code is identical to that shown in the description of IJROT(J).
IKTRN(J)	Integer	<u>i - k</u> <u>translation</u> . Code number for the translational constraint between beam members in the "i" and "k" positions at the J th joint, unitless. The code is identical to that shown in the description of IJROT(J).
ILTRN(J)	Integer	<u>i - l</u> <u>translation</u> . Code number for the translational constraint between beam members in the "i" and "l" positions at the J th joint, unitless. The code is identical to that shown in the description of IJROT(J).
IOJTRN(J)	Integer	<u>i or j</u> <u>translation</u> . Code number for the translational constraint between the ground and beam member in either the "i" or "j" position at the J th joint, unitless. The code is identical to that shown in the description IJROT(J).
ISID	Logical	<u>input stations for input data?</u> True-false indicator to call for printing the physical characteristics at the stations which were input. T = Print only at input stations. F = Print at all stations.
ISOD	Logical	<u>input stations for output data?</u> True-false indicator to call for output of the results of the program. T = Output and plots results at input stations. F = Output and plots results at all stations.
JG	Decimal	Torsional stiffness at the N th station, lb/in ² .
JGMOD	Decimal	JG modification. Constant by which each value of JG(N) will be multiplied, unitless.
MASMOD	Decimal	<u>mass modification</u> . Constant by which each value of MASS(N) will be multiplied, unitless.

<u>Name</u>	<u>Type</u>	<u>Description</u>
MASS(N)	Decimal	<u>mass</u> . Mass per unit length at N th station, lb.sec ² /in. ² .
MBRI(J)	Integer	<u>member i</u> . Identification number of the beam member in position "i" at the J th joint, unitless.
MBRJ(J)	Integer	<u>member j</u> . Identification number of the beam member in position "j" at the J th joint, unitless.
MBRK(J)	Integer	<u>member k</u> . Identification number of the beam member in position "k" at the J th joint, unitless.
MBRL(J)	Integer	<u>member l</u> . Identification number of the beam member in position "l" at the J th joint, unitless.
NOMODE	Integer	<u>no. of modes</u> . Total number of modes to be computed, unitless. If negative, the values of the frequency and U determinant from the search routine will be printed.
NON	Integer	<u>no. of N's</u> . Total number of stations on a beam member, unitless.
NORDIS	Logical	<u>normalize to displacement?</u> True-false indicator to call for normalization of each eigenvector to its station of maximum modal displacement. T = Normalize to maximum displacement. F = Do not normalize to maximum displace- ment.
NOREQM	Logical	<u>normalize to equivalent mass?</u> True-false indicator to call for normalization of each eigenvector to its associated generalized mass. T = Normalize to unit generalized mass. F = Do not normalize to unit generalized mass.
NORMBR	Integer	<u>normalize on member</u> . Identification number of beam member on which the normalization station is located, unitless.
NORM	Integer	<u>normalize on station N</u> . Subscript of input station on beam member NORMBR at which to normalize the eigenvectors of every mode, unitless.

<u>Name</u>	<u>Type</u>	<u>Description</u>
OMEG(I)	Decimal	<u>omega</u> . Array of frequencies accepted by the program as known modal frequencies, rad./sec.
OMEGA	Decimal	<u>omega</u> . First trial frequency to be used in the search for natural frequencies, rad./sec. OMEGA should be less than the expected first frequency.
OMEGA SUBR	Decimal	<u>omega_r</u> . Reference value of frequency. rad./sec.
OMGTOL	Decimal	<u>omega tolerance</u> . Relative accuracy criteria for the natural frequency convergence, unitless.
PLOT	Logical	<u>plot</u> . True-false indicator to call for Calcomp plot. Initialized as F. T = Plot. F = Do not plot.
RFAE	Decimal	<u>reference of AE</u> . Reference value of axial stiffness, lb. in. ² .
RFJG	Decimal	<u>reference of JG</u> , lb. in. ² .
RFMASS	Decimal	<u>reference of mass</u> . Reference value of mass per unit length, lb. sec. ² /in.
RFX	Decimal	<u>reference of X</u> . Reference value of lengthwise coordinate, in.
RFZR	Decimal	<u>reference of ZR</u> . Reference value of mass moment of inertia. lb.sec. ²
TENS	Integer	<u>tension column</u> . Identification number for the column in the output in which the TENS(N) will be printed out.
TENSION	Decimal	<u>tension</u> . Axial force at a station, lbs.
TENSION-PRIM	Decimal	longitudinal strain at a station lb./in.
TENS(N)	Decimal	<u>tension</u> , axial force at the Nth station, lb. (before normalization).
TIME	Decimal	<u>time</u> . Value of time associated with each group of output data, seconds. (TIME is not used in the program).

<u>Name</u>	<u>Type</u>	<u>Description</u>
TRNFIJ(J)	Decimal	<u>translational flexibility constant between i and j.</u> Translational flexibility constant between beam members in positions "i" and "j" at the J th joint, in./lb.
TRNFIK(J)	Decimal	<u>translational flexibility constant between i and k.</u> Translational flexibility constant between beam members in positions "i" and "k" at the J th joint, in./lb.
TRNFIL(J)	Decimal	<u>translational flexibility constant between i and l.</u> Translational flexibility constant between beam members in positions "i" and "l" at the J th joint, in./lb.
TRNS(J)	Decimal	<u>translational spring constant.</u> Translational spring constant between the ground and beam member in either the "i" or "j" position at the J th joint, lb./in.
X(N)	Decimal	<u>X.</u> Lengthwise position coordinate of the N th station, in.
XMOD	Decimal	<u>X modification.</u> Constant by which each value of X(N) will be multiplied, unitless.
ZETA(N)	Decimal	<u>zeta.</u> Modal displacement at N th station, in. (before normalization).
ZETA	Decimal	longitudinal or torsional deflection at a station, unitless.
ZETA PRIME	Decimal	slope of longitudinal or torsional deflection at a station, unitless.
ZR	Decimal	mass moment of inertia at N th station, lb.sec. ²
ZZAC	Integer	<u>zeta column.</u> Identification number for the column in the output in which ZETA(N) will be printed, unitless.
TORQ	Integer	<u>torque column.</u> Identification number for the column in the output in which the torque will be printed, unitless.
TORQUE	Decimal	<u>torque.</u> Torsional force at a station, in./lb.
TORQUE-PRIME	Decimal	<u>torque-prime.</u> Rate of change of torsional force at a station, lb.

B. Input Plot Variable Names

<u>Name</u>	<u>Type</u>	<u>Description</u>
INCHX	Integer	Number of inches for the horizontal scale (x).
XMIN	Decimal	Minimum value of X, to appear at lower left hand corner of graph.
DX	Decimal	Δx , per inch of graph.

The following variables are set to 0 in the program and need only be entered if their value should be 1.

PLTM	Integer	= 1, plot mass = 0, no mass plot
PLTZR	Integer	= 1, plot roll inertia = 0, no roll inertia plot
PLTAE	Integer	= 1, plot axial stiffness coefficient = 0, no axial stiffness coefficient plot
PLTJG	Integer	= 1, plot torsional stiffness coefficient = 0, no torsional stiffness coefficient plot
PLTZ	Integer	= 1, plot ζ = 0, no ζ plot
PLTZPR	Integer	= 1, plot ζ' = 0, no ζ' plot
PLTT	Integer	= 1, plot tension or torque curves = 0, no tension or torque plots
MPLT	Integer	= 1, torque (tension) plots for each of 3 modes will be drawn on separate grids = 0, all 3 modes of the torque (tension) plots will be drawn on the same grid.

The following variables are set to 1 in the program and need only be entered if their value should be 2.

NMZR	Integer	= 1, use ascale subroutine to find appropriate scale values for mass or roll inertia plot = 2, read in mass minimum and Δ mass or roll inertia minimum and Δ roll inertia
------	---------	--

<u>Name</u>	<u>Type</u>	<u>Description</u>
NMAEJG	Integer	= 1, use Ascale subroutine to find appropriate scale values for axial or torsional stiffness coefficient plot = 2, read in axial or torsional stiffness scale values
NZAPR	Integer	= 1, use Ascale subroutine to find appropriate scale values for ζ' plots = 2, read in ζ' scale values
NPLTT	Integer	= 1, use Ascale subroutine to find appropriate scale values for torque (or tension) plots = 2, read in torque (or tension) scale values

If NMZR = 2, input the following:

MZRMIN	Decimal	minimum scale value for mass (or roll inertia)
DMZR	Decimal	Δ mass (or Δ roll inertia) value, per inch, total 4 inch grid

If NMAEJG = 2, input the following:

AJMIN	Decimal	minimum scale value of the axial (or torsional) stiffness coefficient
DAEJG	Decimal	Axial (or torsional) stiffness coefficient, per inch, total 4 inch grid

If NZAPR = 2, input the following:

ZPMIN1	Decimal	minimum scale value of ζ'_1
DZP1	Decimal	$\Delta\zeta'_1$, scale value per 1/2 inch, total 1 inch grid
ZPMIN2	Decimal	minimum scale value of ζ'_2
DZP2	Decimal	$\Delta\zeta'_2$, scale value per 1/2 inch, total 1 inch grid
ZPMIN3	Decimal	minimum scale value of ζ'_3
DZP3	Decimal	$\Delta\zeta'_3$, scale value per 1/2 inch, total, 1 inch grid

If NPLTT = 2, input the following:

<u>Name</u>	<u>Type</u>	<u>Description</u>
TMIN	Decimal	minimum scale value of torque (tension)
DTT	Decimal	Δtorque (Δtension) scale value per inch, total 4 inch grid

OUTPUT

Printed

After the title cards are printed, the input controls data is listed. If ISID = T, the physical characteristic at only the input stations are printed. If ISID = F, the physical characteristics at all stations are printed. The total mass, static moment, and center of gravity for the branched beam are printed for both the longitudinal and torsional vibration analysis.

An option has been made available to print the non-dimensionalized frequency and the corresponding determinant values for [U] throughout the iteration process. This print is requested by inputting a minus sign in front of the desired nomode value, i.e., NOMODE = -3,

IF ISOD = T, the modal frequency, generalized mass, and station properties are printed at each input station only. If ISOD = F, the modal frequency, generalized mass, and station properties are printed at all stations.

Plotted

If PLOT = F, or is not entered, no plots are made. One or more variables may be plotted against X on the Calcomp plotter if PLOT = T. A header card and \$NAM1 namelist must be read into program if plot = T. See figure 7 for a sample of the request card for the plots. Also, if PLOT = T, a statement is printed as each plot is completed.

DIMENSIONALIZATION

The following equations are used in the nondimensionalization and modification of the input values.

```
X = (X)(XMOD)/(RFX)
MASS = (MASS)(MASMOD)/(RFMASS)
AE = (AE)(AEMOD)/(RFAE)
JG = (JG)(JGMOD)/(RFJG)
ZR = (ZR)(ZRMOD)/(RFZR)
OMEGA = (OMEGA)(RFX)(RFMASS)1/2/(RFAE)1/2
DELOMG = (DELOMG)(RFX)(RFMASS)1/2/(RFAE)1/2
TRNS = (TRNS)(RFX)/(RFAE)
TRNFIJ = (TRNFIJ)(RFAE)/(RFX)
TRNFIK = (TRNFIK)(RFAE)/(RFX)
TRNFIIL = (TRNFIIL)(RFAE)/(RFX)
```

DISCUSSION OF COMPUTER PROGRAM

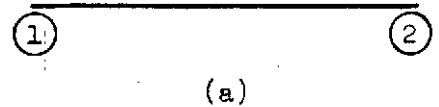
As described in the Application of the Method to Torsional Vibrations, the equations for the torsional vibration analysis are analogous to the equations for longitudinal vibrations.

The namelist method of input is used in this program, see CDC 6600 Computer Systems Fortran Reference Manual for a description. The namelist \$LONVIB is used to input the modal data for the longitudinal analysis. The namelist \$TORVIB is used to input the modal data for the torsional analysis. Since the two analyses are analogous, the corresponding inputs of the two namelists are equivalenced.

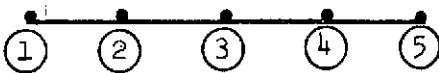
The input to this program consists of the necessary physical characteristics of the structure, definition of the boundary constraints, and controls to define options. In order to assist the user, it is necessary to elaborate on some of these terms.

Beam.- The vehicle or structure under analysis is composed of a system of beams or "members". Because of the program's ability to analyze branched beam problems, the basic structure characterized by a straight continuous elastic axis is referred to as a main beam; appendages which are attached to the main beam having elastic axes parallel to the main beam are then referred to as branches.

Joint.- The ends of the members are called joints. The word "joint" describes the beginning and the end of a member between which the number of stations are used to describe the physical characteristics of a member. A uniform continuous beam may be described by a single member having two ends and therefore, "two joints", or it may be described by any number of members having the appropriate number of joints, with the relationship between joints described by translational constraints. For example, two identical beams are represented in sketch 14(a) and (b).



(a)



(b)

Sketch 14.- Beam-joint arrangement.

Beam (a) consists of a single member with joints (1) and (2), the boundary conditions at the end are considered to be free-free. Joint 1 then has only a "j" member referring to sketch 14. Joint 2 has only an "i" member. In sketch 14(b) the same beam may be represented by four members with the extremes of the beam having the same boundary conditions. Joint 1 again has only a "j" member. Joint 2 has two members, 1 and 2, with member 1 the "i" member and member 2 the "j" member. Because (a) and (b) are identical uniform beams, members "i" and "j" at joint 2 are fully constrained as well as joints 3 and 4 with joint 5 identical to joint 2 of (a). It can be seen that the characteristics of both beams are identical. There are many uses for dividing beams into several members. For example if a change in the physical characteristics were required in the second quarter of the beam in figure 14 it could only be necessary to describe member 2 in sketch 14(b). If, however, the beam as described in sketch 14(a) were utilized then the entire beam input would have to be reentered.

Physical characteristics.-The physical characteristics necessary to compute the natural longitudinal modal data are entered as a continuous system and include mass and axial stiffness. The physical characteristics necessary to compute the natural torsional modal data include the torsional stiffness and the polar mass moment of inertia.

Output.- First the input data is printed, then the total mass, center of gravity, and the rigid body static moment of the beam being analyzed are printed. Also the generalized mass for each mode is printed for the longitudinal case. Next, the modal frequencies, both in cycles per second and radians per second are printed. Finally, at each station, four of the modal characteristics are printed.

Test problem.- An idealized beam, as shown in sketches 11 and 12 will be used to exercise the subject program. The necessary physical characteristics are provided. Two separate cases will be run. First the longitudinal modal data will be computed. The data required will be the first five longitudinal natural frequencies, mode shapes, axial forces, and the first derivatives of the axial forces, and mode shapes. Also, the static moment, center of gravity and total mass of the beam are required. In addition, the generalized mass is required for each mode.

In the second case, the torsional modal data will be calculated. The data required will be the first five torsional modes, torques, and their derivatives. The static moment, center of gravity, and total mass of the beam are also required. The generalized mass is not output for the torsional modes.

Identification.- Any number of cards may be used to identify or describe the problem with a minimum of one. However, the first card will be used as a title on each page of the printout. The contents of all of the title cards will be printed at the beginning of the printout.

Data.- Immediately following the identification cards are cards containing the physical characteristics of the problem. For the longitudinal case each set of data for each beam member begins and ends with \$LONVIB and contains; first, TIME, which is an identifying time; second, NON, the number of stations of input for the member; third, BEAM, a number identifying the member; and fourth, the characteristics of the member X(N), MASS(N), AE(N). The data for each member is entered in the order established in sketch 12. Although the beam is numbered in the data, the program numbers the members in the order in which they are entered.

The input for the torsional vibration data is identical to the longitudinal vibration case with the exception that \$TORVIB is substituted for \$LONVIB, ZR(N) for MASS(N), and JG(N) for AE(N).

Examples:

or

```
$LONVIB  
BEAM = 1, TIME = 0.0, NON = 2,  
X(1) = 0.0, 40.,  
MASS(1) = 0.0009, 0.0009,  
AE(1) = 36815625., 36815625., $  
$LONVIB  
BEAM = 2, TIME = 0.0, NON = 2,  
X(1) = 40., 120.,  
MASS(1) = 0.000144, 000144  
AE(1) = 5890500., 5890500., $  
$LONVIB  
BEAM = 3, TIME = 0.0, NON = 2,  
X(1) = 40., 120.,  
MASS(1) = 0.0009, 0.0009  
AE(1) = 36815625., 36815625., $
```

```
$TORVIB  
BEAM = 1, TIME = 0.0, NON = 2,  
X(1) = 0.0, 40.  
ZR(1) = 0.00220106, 0.00220106  
JG(1) = 6200000., 6200000., $  
$TORVIB  
BEAM = 2, TIME = 0.0, NON = 2  
X(1) = 40., 120.,  
ZR(1) = 0.0000045, 0.0000045  
JG(1) = 76699., 76699., $  
$TORVIB  
BEAM = 3, TIME = 0.0, NON = 3,  
X(1) = 40., 120.,  
ZR(1) = 0.00022, 0.00022,  
JG(1) = 6200000., 6200000., $
```

Joint description.- The next group of data informs the program of the proper assembly of members that have been input, describes the constraint units at each joint, and essentially builds the mathematical model of the beamlike structure. For the test problem, the relation holds as shown in sketch 12. The test problem member and the joint relationship is shown with joints listed in the first column and the associated members tabulated as illustrated in Table IV.

JOINT	MBRI	MBRJ	MBRK	MBRL
1	0	1	0	0
2	1	3	0	2
3	2	0	0	0
4	3	0	0	0

TABLE IV

Joint Arrangement

The next relationship to be shown is the listing of the constraints between members at their joints. The constraints for the test problem are illustrated in Table V.

JOINT	IOJTRN	IJTRN	IKTRN	ILTRN
1	3	0	0	0
2	3	1	0	2
3	3	0	0	0
4	3	0	0	0

TABLE V

Joint Constraints

The codes for constraints are; 1 = fully constrained or fixed, 2 = partially constrained, 3 = no constraint or free. If no relationship exists a zero is entered. The second column describes the constraint between either the i or j member and the ground. The test beam has no constraint to ground at any joint, and therefore code 3 is entered. Column 3 describes the translational constraint between members i and j at each joint; similarly the fourth column describes the constraint between i and k and the fifth column describes the constraint between i and l. If no relationship exists, then zero is entered.

The remaining data necessary to complete the joint description are the values for the partial constraints to indicate the degree of the constraint. The tabulation is shown in Table VI for the test beam.

JOINT	TRNS	TRNFIJ	TRNFIK	TRNFIL
1	0	0	0	0
2	0	0	0	1×10^{-5}
3	0	0	0	0
4	0	0	0	0

TABLE VI

Joint Partial Constraint Coefficients

This Data group is entered as \$JOINT and is input once for each problem. All data shown in Table I must be entered. All data shown in Tables IV and V must also be entered except where the data is zero.

Example:

```
$JOINT
MBRI(1) = 0, 1, 2, 3,
MBRJ(1) = 1, 3, 0, 0,
MBRK(1) = 4*0,
MBRL(1) = 4*0,
IOTRN(1) = 4*3,
IBTRN(1) = 0, 1, 0, 0,
IKTRN(1) = 4*0,
ILTRN(1) = 0, 2, 0, 0,
TRNS(1) = 4*0,
TRNFIJ(1) = 4*0,
TRNFIK(1) = 4*0,
TRNFIL(1) = 0, 1.x10-5,0, 0,
```

Controls.— Further input is necessary to control the computer in its solution of the modal data. The first part of the control input is called \$CTRL. It modifies the input parameters. The input \$LONVIB or \$STORVIB which is originally dimensional becomes non-dimensional through controls in the program.

The reference mass, RFMASS, the reference length, RFX, and the reference axial stiffness coefficient, RFAE should be set = to 1.0. Non-dimensionalization of the physical characteristics was necessary in the lateral vibration program because of the combination of very large and very small terms in the frequency determinant. However, this situation does not exist for the longitudinal and torsional vibration program.

The second function of the \$CTRL group is to provide parameter modifiers. This group contains a group of variable multipliers that operate on the input physical characteristics before they are used in the program. These modifiers may be used to convert the units of input to the desired units. Also, since tables of stiffness coefficients are occasionally premultiplied by a constant to simplify the tabulations, these may also be entered and the proper modifier entered to correct the tabulated characteristics. These modifiers are entered only when not equal to one.

The final item of input in this group is DELX. This variable introduces additional stations in areas where the distance between stations of input is greater than desired for accuracy. The equations in the program are based on linear change of modal characteristics between stations along the length of the beam. Also the lengthwise integrations require sufficient stations in order to be able to assume linearity between steps. A value for DELX equal to 1/100 of the total length of the beam is usually adequate, and stations will be added only when the value of $X_{n+1} - X_n > \text{DELX}$, with linear interpolation of physical characteristics for the added station values. This completes the \$CTRL group, and an example follows:

Example:

```
$CTRL
RFX = 1.0, RFMASS = 1.0, MASMOD = .01
RFAE = 1.0, AEMOD = 1.E8, DELX = 2.0, $
```

The next group of input is called \$MDES and provides the information necessary to calculate the natural frequencies. The frequency at which the search routine will start is OMEGA; the increment of ω in the routine, DELOMG; the tolerance at which the routine is considered to have solved for a natural frequency, OMGTOL; the number of modes to be solved, NOMODE; the mode number to be assigned to the first mode of input, FRSTMD; and finally the normalizing criteria, NORN, NORMBR, NOREQM.

The use of OMEGA to start the frequency search routine serves two purposes. First, the routine should be started somewhat lower than an estimated fundamental frequency to make optimum use of the computer time, especially in very stiff beams where the fundamental frequency might be high. Second, it allows the frequency search routine to start at a higher mode than the fundamental without calculating lower modes. OMGTOL is the relative difference in trial frequencies, used in the frequency search routine. OMGTOL is normally set at $1.x10^{-6}$, giving excellent results with optimum use of computer time. FRSTMD is used only when the frequency search routine is asked to start at other than the fundamental mode. Normally, FRSTMD is set internally equal to 1 and titles the first mode of output as "Mode 1" with arithmetic progression of successive modes of output.

The program has the capability of normalizing to unity at either the station of maximum deflection or at any particular station on the beam. Also it can normalize the mode shape so that the generalized mass is equal to unity. To normalize at maximum displacement equal to unity set NORDIS = T; at generalized mass equal to unity, NOREQM = T; at a particular station equal to unity, NORMBR = member containing the station, and NORN = number of the station in the member. The data for the test case is shown in the following example.

Example:

```
$MODES
OMEGA = 25., DELOMG = 200., OMGTOL = 1.E-7,
NOMODE = 5, NORDIS = T, $
```

The final group of input describes the output desired. The output will contain a complete listing of input, complete in such detail that the program may be reconstructed from this data alone. The physical characteristics are arranged in the proper size with headings for direct integration into reports. The stations are listed in a double set of integers with the first integer identifying the member and the second the station of the member. Immediately following the physical characteristics, the center of gravity, the total mass, and the rigid body static moment of the problem are printed.

The last item of output contains the modal characteristics as computed in the subject program. Each page of this modal data contains the contents of the first identification card, and also the number of the mode. The output, on the first page for each mode, also lists the natural longitudinal or torsional frequencies of the mode in both cycles per second and radians per second. In addition, the generalized mass for the normalized mode shape is printed for the longitudinal case.

Following this information are six columns of modal data. The first two are always the station number and its lengthwise location with respect to the established origin, respectively. The next four columns are the modal characteristics. This type of output is repeated for the required number of modes. All output may be printed at only the points of input or may include the additional stations provided by DELX. Additional outputs are a plot option which gives Calcomp plots.

This final group of data is called \$OUTPUT. Four of the modal characteristics will be output. ISID calls for printing the input characteristics at either the original stations or including the stations added by DELX, and similarly, ISOD controls the output print. Both are set to F internally and need to be entered if only the input stations are to be printed. If only stations of input are desired set ISID = T, and if all the stations are desired for output set ISOD = F. In the same manner, if plots are desired, set PLOT = T. The identification for the column of output assigned to the modal characteristics does not include the first two columns which are always "station" and "x". The following example applies to the test problem.

Example:

```
$OUTPUT
ZTAC = 1, DZTAC = 2, TENS = 3, DTENS = 4,
ISID = T, ISOD = F, $
```

This data states that after each station and its appropriate "X" location the first column will be deflection, zeta (ZTAC = 1); the second column will be the first derivative of the deflection, (DZTAC = 2); the third column will be the axial force, (TENS = 3) or for torsional vibration the torque, (TORQ = 3); and the fourth column will be the first derivative of the axial force (DTENS = 4) or the first derivative of the torque, (DTORQ = 4). The input is terminated with \$END.

This concludes the formal presentation of input required for this portion of the program. The second part of the program is designed to plot upon request any or all of the following depending upon the analyses: mass or roll inertia; stiffness coefficient, deflection or rotation; axial force or torque; and the corresponding first derivatives. The plots will be drawn for the first modes of the analysis. If plot = T in the \$OUTPUT group then \$NAM1 must be read. This group includes all of the plot options and the appropriate scales to be used in the plots.

Concluding remarks on computer program.-- It should be emphasized that due to the assumption of linearity of the system variables over the length X_n to X_{n+1} , this increment must be small. Though studies conducted of comparing the frequency accuracy as a function of station separation, it has been found that 1/100 of the length of the beam gives very good results for the first five modes. Another benefit derived from the additional stations inserted in spans of constant parameters is in the quality of the plots of output characteristics with the additional points. The total number of added stations due to DELX cannot exceed 600. The total number of branches and beams cannot exceed 20.

A misleading situation may occur when values of AE or JG are near zero. This situation may occur at the free end of beams. The problem manifests itself in a radical variation of the mode shapes in the area approaching the upper boundary of the span. This variation is a combined result of near zero stiffness of AE and/or JG and the failure to achieve numerically the absolute zero. This condition is readily corrected by avoiding near the free ends, AE and JG values of less than 0.0001 of their respective average over the total span.

The output will be only as good as the math model representing the problem. With a proper input, the user can expect frequencies of the lower modes of vibration to be accurate within less than 1/2 of one percent. Some feeling for the accuracy may be gained by investigating the modal characteristics at the boundaries. The absolute value of zero for some boundary conditions of free-free beams cannot be achieved by digital computers, therefore the analyst should accept finite boundary values that are a fraction of 1 percent of the peak

absolute value of their respective functions over the length of the beam. Another suggestion for gaining confidence in the modal data is to observe the continuity of the displacements and the axial forces or torques in areas where branches are attached to main beams.

APPENDIX B

COMPUTER PROGRAM LISTING

```

OVERLAY(LINK,0,0)
PROGRAM TORLONG(INPUT=201,OUTPUT=201,TAPE5=INPUT,TAPE6=OUTPUT,TAPE
12=201,TAPE3,TAPE4=201)
COMMON NPLT,ENDFILS
INTEGER ENDFILS
NSUM=0
ENDFILS=1
NPLT=0
LINK=4LLINK
CALL CALCOMP
CALL LEROY
1 CALL OVERLAY (LINK,1,0,0)
IF (ENDFILE.EQ.0) 2,3
2 IF (NSUM.GT.0) WRITE (6,4) NSUM
CALL CALPLT (0.,0.,999)
STOP
3 CALL OVERLAY (LINK,2,0,0)
NSUM=NSUM+NPLT
GO TO 1
C
4 FORMAT (/24H TOTAL NUMBER OF PLOTS =,I5)
END

```

```

OVERLAY(LINK,1,0)
PROGRAM VTBRAT
C
000003      TITLE      BRANCH BEAM TORSIONAL AND LONGITUDINAL VIBRATIONS ANALYS      B  1
              DIMENSION TITLE(28), HEADNG(2,4), TITLES(2,6), EA(600), STN(600),
              ISSAN(600), HI(600), DY(2,600), Y(2,600), TITL(12), FORM(8), VARNO(24),
              MASS(600), NUM(20), MBRK(30), I0JTRN(30), MBRI(30), LV(6), MBR3L(30),
              IJTRN(30), MBRJ(30), MV(6), SAM(10), ILTRN(30), IKTPN(30),
              &OMFG(12), FORMA(24), OMFORM(7)      B  2
000003      DTMHENSION AE(600), ZR(600), JG(600), ARRY(3), LARRY(3), TARRY(3)      B  3
000003      COMMON NPLOT,ENDFILS      B  4
000003      INTEGER ENDFILS      B  5
000003      COMMON /BLK1/ JC(30,8),TRNS(30),TRNFI(J(30),TRNFIK(30),TRNFI(30),J      B  6
1NT
000003      COMMON /BLK2/ X(600),HH(600),MAS(600),NSTA(30)      B  7
000003      COMMON /BLK3/ U(100,60),B(2,2,30),OMEGA,NOMBR,TEMP(2,30),GMASS.      B  8
000003      COMMON /BLK4/ UDET(20),NU,DELOMG,OMGTOL,IT,ITER,NUCT      B  9
000003      COMMON /BLK5/ NFRQ,NTROL(13),FRSTMD      B 10
000003      INTEGER ZTAC,OZTAC,TENS,DTENS,FRSTMD,TOPO,DTORQ      B 11
000003      REAL MASS,NCMR,MAS,MASMOD,JG,JGMOD,LARRY      B 12
000003      LOGICAL ISID,TSOD,NORDIS,NOREQM,PLCT,TORVB      B 13
000003      EQUIVALENCE (EA,HH,AE,JG), (RFAE,PFE4,PFJG), (EAMOD,AEMOD,JGMOD),
1(PFMASS,PFZR), (MASMOD,ZRMOD), (TENS,TORQ), (DTENS,DTORQ), (MASS,M
2AS,ZR)      B 14
000003      EQUIVALENCE (MBRI(1),JC(1,1)), (MBRJ(1),JC(1,2)), (MBRK(1),JC(1,3)
1), (MBRL(1),JC(1,4)), (I0JTRN(1),JC(1,5)), (IJTRN(1),JC(1,6)), (IK
2TRN(1),JC(1,7)), (ILTPN(1),JC(1,8))      B 15
000003      EQUIVALENCE (U,Y), (U(1,13),DY), (U(1,25),STN), (U(1,31),SSAM), (U
1(1,37),HI)      B 16
000003      DATA DOLLAR/2H $/,F/3H F/,T/3H T/      B 17
C
000003      DATA (TITL(I),I=1,8)/6H0NV18,6HTORVIB,6HJOINT ,6HCTRL,6HINPFRQ,
16HMODES ,6HOUTPUT,6HEND /
000003      DATA FORM/30H(5X,I3,1H-,I3,   OPF10.3,OP  /,FORM(5)/6HE14.5)/,(VA
1RN0(I),I=1,4)/2H 1,2H 2,2H 3,2H 4/,BLANK/6H  /,RFJ/6HRFJG =/,R
2FZ/8H RFZR =/,RFA/6HRFAE =/,PFM/8HRFMASS =/,PNORE/9HNOREQM = /,ZR
3M/8H ZRMOD =/,ASM/BHMASHMOD =/,AEM/7HAEMOD =/,GJM/7HJGMOD =/,LARRY/
430H FOR LONGITUDINAL VIBRATIONS /,TARRY/30H FOR TORSIONAL VIBRATI
SONS  /,ZRIN/4H ZR/,GJIN/2HJG/,RMINP/4HMASS/,RAE/2HAE/
C
000003      DATA TITLES(1,1)/6H    TEN/,TITLES(1,2)/6H    Z/,TITLES(1,3)/6HTEN
1ST0/,TITLES(1,4)/6H ZETA-/ ,TITLES(2,1)/4HSION/,T.TLES(2,2)/3HETA/ ,
2TITLES(2,3)/6HN-PRIM/,TITLES(2,4)/5HPRIME/,TITLES(1,5)/6H    TOR/,T

```

	3 TITLES(2,5)/3HQUE/, TITLES(1,6)/6HTORQUE/, TITLES(2,6)/6H-PRIME/	B 42
C	DATA FORMA/260H(IH 54X,8HTABLE II	B 43
000003	1 //40XA2,13A6//50X3HT =F8.2,4H	B 44
	2 SEC //56X,4HMODE13//5X,28HFREQUENCY CYC	B 45
	3LES PER SECOND F12.3//5X,28HFREQUENCY RADIANS PER SECOND F12.5,	B 46
	4 / /	B 47
C	DATA OMFORM/70HF12.8 ,F12.7 ,F12.6 ,F12.5 ,F12.4 ,F	B 48
000003	112.3 ,E16.8 ,/	B 49
C	NAMELIST /LONVIB/ X,MASS,AE,NON,TIME,BEAM	B 50
000003	NAMELIST /INPFRC/ OMFG/MODES/NOMOOF,DELCMG,OMGTOL,NORN,NORMBR,NORD	B 51
000003	1IS,NOREQM,OMEGA,FRSTMD	B 52
000003	NAMELIST /JOINT/ TRNS,MBRI,MBRJ,MBRK,MBRL,ICJTRN,IJTRN,IKTRN,ILTRN	B 53
000003	1,TRNFIJ,TRNFIK,TRNFIIL	B 54
000003	NAMELIST /CONTPL/ RFX,RFMASS,RFAE,XMOD,MASMOD,AEMOD,CELX,JGMOD,ZRM	B 55
	10D,RFJG,RFZR	B 56
000003	NAMELIST /OUTPUT/ ZTAC,DZTAC,TENS,DTENS,ISIO,ISOD,PLOT,TORQ,DTORQ	B 57
000003	NAMELIST /TORVIB/ X,ZR,JG,NON,TIME,BEAM	B 58
000003	REWIND 2	B 59
000005	REWIND 4	B 60
000007	1 CONTINUE	B 61
000007	DO 2 I=1,30	B 62
000011	TRNS(I)=0.	B 63
000012	TRNFIJ(I)=0.	B 64
000013	TRNFIK(I)=0.	B 65
000014	TRNFIIL(I)=0.	B 66
000015	2 CONTINUE	B 67
000017	DO 3 I=1,12	B 68
000020	OMEG(I)=0.	B 69
000021	3 CONTINUE	B 70
000023	DO 4 I=1,8	B 71
000024	DO 4 J=1,30	B 72
000025	4 JC(J,I)=0	B 73
000026	K=0	B 74
000037	M=0	B 75
000040	KK=0	B 76
000041	MM=0	B 77
000042	IFPO=0	B 78
000043	ZTAC=0	B 79
000044	DZTAC=0	B 80
000045	TENS=0	B 81
		B 82
		B 83
		B 84

000046	DTENS=0	8 85
000047	NOPRN=0	8 86
000050	NOPMBR=0	8 87
000051	FRSTMID=1	8 88
000052	PLOT=.FALSE.	8 89
000053	TSID=.FALSE.	8 90
000054	TSOD=.FALSE.	8 91
000055	NORDIS=.FALSE.	8 92
000056	NOPEQM=.FALSE.	8 93
000057	RFEA=1.	8 94
000060	RFX=1.	8 95
000061	RFMASS=1.	8 96
000062	EAMOD=1.	8 97
000063	XMOD=1.	8 98
000064	MASMOD=1.	8 99
000065	DELX=1.	8 100
C READ TITLE CARDS	8 101
000066	READ (5,100) (TITLE(I),I=1,14)	8 102
000077	IF (ENDFILE 5) 5,6	8 103
000102	5	8 104
000103	ENDFILE5=0	8 105
000105	RETURN	8 106
000106	5	8 107
000117	WRITE (6,101) (TITLE(I),I=1,14)	8 108
000131	READ (5,100) (TITLE(I),I=15,28)	8 109
000133	IF (DOLLAR.EQ.TITLE(15)) GO TO 8	8 110
000135	WRITE (6,102) (TITLE(I),I=15,28)	8 111
000146	GO TO 7	8 112
000148	8	8 113
000150	BACKSPACE *	8 114
000152	DO 9 I=1,8	8 115
000170	9	8 116
000172	IF (TITLE(16).EQ.TITL(1)) GO TO (10,13,14,18,19,22,23,24), 1	8 117
000170	CONTINUE	8 118
000172	WRITE (6,103) TITLE(15),TITLE(16)	8 119
000202	STOP	8 120
C READ STATION DATA	8 121
000204	10	8 122
000207	READ (5,LONVIB)	8 123
000210	TOPVB=.FALSE.	8 124
000211	M=M+1	8 125
000212	NUM(M)=NON	8 126
C MOVE DATA TO ARRAY BEHIND LAST ONE READ.	8 127
000214	DO 12 L=1,NON	8 128
000215	K=K+1	8 129
000217	STN(K)=X(L)	8 130
000221	HI(K)=EA(L)	8 131
000223	SSAM(K)=MASS(L)	8 132

000225	12	CONTINUE	B 126
000230		NOMBR=M	B 129
000231		GO TO 7	B 130
000232	13	READ (5,TORVIB)	B 131
000235		TORVBS=.TRUE.	B 132
000236		GO TO 11	B 133
000237	14	READ (5,JOINT)	B 134
000242		DO 16 I=1,30	B 135
000244		DO 15 J=1,4	B 136
000245		IF (JC(I,J).NE.0) GO TO 16	B 137
000250	15	CONTINUE	B 138
000252		GO TO 17	B 139
000252	16	CONTINUE	B 140
	C JNT = NO. OF JOINTS	B 141
000254		JNT=30	B 142
000255		GO TO 7	B 143
000256	17	JNT=I-1	B 144
000260		GO TO 7	B 145
000261	18	READ (5,CTRL)	B 146
000264		GO TO 7	B 147
000265	19	READ (5,INPFRQ)	B 148
000270		DO 20 I=1,12	B 149
000272		IF (IMEG(I)) 20,21,20	B 150
000273	20	CONTINUE	B 151
	C IFRQ = NO. OF INPUT FREQUENCIES	B 152
000275		IFPQ=12	B 153
000276		GO TO 7	B 154
000277	21	IFRQ=I-1	B 155
000301		GO TO 7	B 156
000302	22	READ (5,MODES)	B 157
000305		GO TO 7	B 158
000306	23	READ (5,OUTPUT)	B 159
000311		GO TO 7	B 160
	C ALL INPUT IN, BEGIN CALCULATION	3 161
	C READ TO SKIP SEND CARD	B 162
000312	24	READ (5,100) TITLE(15)	B 163
000320		NPPNT=0	B 164
000321		IF (NOMODE.GE.0) GO TO 25	B 165
000323		NOMODE=-NOMODE	B 166
000323		NPPNT=1	B 167
000324	25	MN=0	B 168
000325		DO 26 II=1,NOMBR	B 169
000327	26	MN=MN+NUM(II)	B 170

```

000333      DO 27 II=1,KN          8 171
000334      STN(II)=STN(II)*XMOD   8 172
000336      HY(II)=HI(II)*EAMOD   8 173
000341      SSAM(II)=SSAM(II)*MASMOD 8 174
000343      27 CONTINUE           8 175
000345      L=0                  8 176
000346      K=0                  8 177
000347      I=1
C      .... EXPAND STATION DATA ARRAYS IF NECESSARY 8 178
000350      DO 32 M=1,NOMBR        8 179
000351      L=NUM(M)+L           8 180
000353      28 I=I+1             8 181
000355      K=K+1               8 182
000356      IF (K.GT.600) GO TO 30 8 183
000361      X(K)=STN(I-1)         8 184
000362      HH(K)=HI(I-1)         8 185
000364      MAS(K)=SSAM(I-1)       8 186
000366      IF (I-L) 29,29,31     8 187
000370      29 IF ((STN(I)-X(K)).LE.DELX) GO TO 28 8 188
C      .... INSEPT EXTRA POINT 8 189
000375      K=K+1               8 190
000376      IF (K.GT.600) GO TO 30 8 191
000401      X(K)=X(K-1)+DELX    8 192
C      .... LINEAR INTEPPOLATION FOR ADDITIONAL VALUES 8 193
000403      PART=(X(K)-STN(I-1))/(STN(I)-STN(I-1)) 8 194
000411      HH(K)=HI(I-1)+(HI(I)-HI(I-1))*PART 8 195
000416      MAS(K)=SSAM(I-1)+(SSAM(I)-SSAM(I-1))*PART 8 196
000424      GO TO 29             8 197
000424      30 WRITE (6,104)        8 198
000430      GO TO 1              8 199
C      .... NSTA ARRAY HAS CUMULATIVE NO. OF STATIONS PER BEAM 8 200
000431      31 NSTA(M)=K          8 201
000433      32 CONTINUE           8 202
C
C      .... CALCULATE THE CENTER OF GRAVITY 8 203
000436      33 CALL CGRAV (NOMBR,TOTM,TOTS,CG) 8 204
C      .... PRINT OUT INPUT 8 205
000441      34 IF (.NOT.ISOD) GO TO 35 8 206
C      .... INPUT REQUIRED FOR FINAL OUTPUT, SAVE ON SCRATCH TAPE 8 207
000443      REWIND 2             8 208
000445      WRITE (2) K,ISTN(I),SSAM(I),HI(I),I=1,K 8 209
000446      END FILE 2           8 210
000467      35 WRITE (6,105) (TITLE(I),I=1,14) 8 211
                                         8 212
                                         8 213

```

000501	C COMPUTE REFERENCE VALUE FOR FREQUENCY	B 214
	C	OMR=SQRT(RFEA/(RFMMASS*RFX**2))	B 215
	C	SET PRINT STATEMENTS	B 216
	C		B 217
000507		PRINT1=F	B 218
000510		PRINT2=F	B 219
000511		IF (NORDIS) PRINT1=T	B 220
000514		IF (TORVB) 36,38	B 221
000516	36	RE=RFA	B 222
000520		RM=RFM	B 223
000521		PRNT2=BLANK	B 224
000523		PRINT2=BLANK	B 225
000524		RMMOD=ZPM	B 226
000525		RAMOD=GJM	B 227
000527		RMAS=ZRIN	B 228
000530		RAEP=GJIN	B 229
000532		DO 37 I=1,3	B 230
000533	37	ARRY(I)=TARRY(I)	B 231
000536		GO TO 40	B 232
000537	38	IF (NOREQM) PRINT2=T	B 233
000542		RE=RFA	B 234
000544		RM=RFM	B 235
000545		PRNT2=PNORE	B 236
000547		RMMOD=ASM	B 237
000550		RAMOD=AEM	B 238
000552		RMAS=RMINP	B 239
000553		RAEP=RAE	B 240
000555		DO 39 I=1,3	B 241
000556	39	ARRY(I)=LARRY(I)	B 242
000561	40	WRITE (6,106) RE,RFEA,NOMODE,RM,RFMMASS,NORMBR,RFX,NORN,OMR,PRINT1, 1DELX,PRNT2,PRINT2,OMEGA,RMMOD,MASMOD,DELCMG,RAMOD,EAMOD,OMGTOL	B 243
000635		WRITE (6,107) (J,(JC(J,I),I=1,8),J=1,JNT)	B 244
000656		WRITE (6,108) (J,(JC(J,I),I=1,4),TRNS(J),TRNFIJ(J),TRNFI 1L(J),J=1,JNT)	B 245
000707		LINE=A	B 246
000710		N=0	B 247
000711		NN=0	B 248
000712		WRITE (6,119)	B 249
000715		WRITE (6,109) ARRY,(TITLE(I),I=1,14),RMAS,RAEP	B 250
000735		I=0	B 251
000736		LMAX=34	B 252
000737		DO 46 M=1,NOMBR	B 253
			B 254
			B 255
			B 256

000741	41	LINF=LINE+1	B 257
000743		IF (LINE.LT.LMAX) GO TO 42	B 258
000745		LINE=0	B 259
000746		WRITE (6,119)	B 260
000751		WRITE (6,110) RMAS,RAEP	B 261
000761	42	N=N+1	B 262
000763		I=I+1	B 263
000764		IF (.NOT.ISID) GO TO 43	B 264
	C PRINT OUT INPUT FOR INPUT STATIONS ONLY	B 255
000765		WRITE (6,111) M,N,STN(I),SSAM(I),MH(I)	B 266
001003		IF (N-NUM(M)) 41,45,45	B 267
	C PRINT OUT INPUT FOR ALL STATIONS	B 258
001007	43	WRITE (6,112) M,N,X(I),MAS(I),HH(I)	B 269
001025		IF (N-NSTA(M)+NN) 41,44,44	B 270
001031	44	NN=NSTA(M)	B 271
001033	45	N=0	B 272
001034	46	CONTINUE	B 273
	C		B 274
001037	47	WRITE (6,113) CG,TOTH,TOTS	B 275
	C SET UP TITLES FOR TABLE IT OUTPUT	B 276
	C CAN PRINT FROM 1 TO 6 COLUMNS OF DATA (INVAR)	B 277
	C SUBSCRIPTS ARE CHOSEN SO THAT ALL OUTPUT IS PRINTED IN	B 278
	C PREFERENCE TO THE U ARRAY (SEE EQUIVALENCE STATEMENTS)	B 279
001051	48	DO 49 I=1,4	B 280
001053		HEADNG(1,I)=BLANK	B 281
001055		HEADNG(2,I)=BLANK	B 282
001057	49	CONTINUE	B 283
001060		NVAR=TENS	B 284
001062		IF (TENS) 54,54,50	B 285
001063	50	IF (TORVB) 51,52	B 286
001065	51	HEADNG(1,TENS)=TITLES(1,5)	B 287
001070		HEADNG(2,TENS)=TITLES(2,5)	B 288
001072		GO TO 53	B 289
001072	52	HEADNG(1,TENS)=TITLES(1,1)	B 290
001075		HEADNG(2,TENS)=TITLES(2,1)	B 291
001077	53	LV(TENS)=1	B 292
001101		MV(TENS)=1	B 293
001103	54	TF (ZTAC) 56,56,55	B 294
001105	55	HEADNG(1,ZTAC)=TITLES(1,2)	B 295
001110		HEADNG(2,ZTAC)=TITLES(2,2)	B 296
001112		LV(ZTAC)=2	B 297
001113		MV(ZTAC)=1	B 298
001115		IF (ZTAC.LT.NVAR) GO TO 56	B 299

001117	NVAR=ZTAC	8 300
001117	56 IF (DTENS) 61,61,57	8 301
001121	57 IF (TOPVB) 58,59	8 302
001123	58 HEADNG(1,DTENS)=TITLES(1,6)	8 303
001126	HEADNG(2,DTENS)=TITLES(2,6)	8 304
001129	GO TO 60	8 305
001130	59 HEADNG(1,DTENS)=TITLES(1,3)	8 306
001133	HEADNG(2,DTENS)=TITLES(2,3)	8 307
001135	60 LV(DTENS)=1	8 308
001137	MV(DTENS)=13	8 309
001141	IF (DTENS.LT.NVAR) GO TO 61	8 310
001143	NVAR=DTFNS	8 311
001143	61 IF (DZTAC) 63,63,62	8 312
001145	62 HEADNG(1,DZTAC)=TITLES(1,4)	8 313
001150	HEADNG(2,DZTAC)=TITLES(2,4)	8 314
001152	LV(DZTAC)=2	8 315
001153	MV(DZTAC)=13	8 316
001155	IF (DZTAC.LT.NVAR) GO TO 63	8 317
001157	NVAR=DZTAC	8 318
C **** PUT RCD FORM OF NVAR IN PRINT FORMAT		8 319
001157	63 FORM(4)=VARNO(NVAR)	8 320
C **** PREPARE TO NON-DIMENSIONALIZE ALL STATION DATA		8 321
001161	XH=RFX/PFXA	8 322
001163	X3H=XH*PFX**2	8 323
001165	HX1=RFFA/RFX	8 324
001166	HX2=HX1/PFX	8 325
001167	HX3=HX2/PFX	8 326
001170	DO 64 I=1,JNT	8 327
001172	TRNS(I)=TRNS(I)*XH	8 328
001174	TRNFIJ(I)=TRNFIJ(I)*HX1	8 329
001176	TRNFIK(I)=TRNFIK(I)*HX1	8 330
001200	TRNFIL(I)=TRNFIL(I)*HX1	8 331
001201	64 CONTINUE	8 332
001203	HCHR=1./PFEA	8 333
001205	MCMR=1/RFMASS	8 334
001207	XCXR=1/PFX	8 335
001212	DO 65 I=1,K	8 336
001213	HH(I)=HH(I)*HCHR	8 337
001215	65 CONTINUE	8 338
001220	DELOMG=DELOMG/CMR	8 339
001221	IF (IFRQ.NE.0) NOMODE=IFRQ	8 340
001223	NOMODE=NOMODE+FRSTMD-1	8 341
001226	TCOUNT=0	8 342

```

001226      IF (PLOT) IXPL=0          8 343
001231      DO 99 NFRQ=FRSTMD,NOMODE 8 344
001233      IF (NPPNT.NE.0) WRTTE (6,114) 8 345
001237      MSTA=NSTA(NOMBR)        8 346
001241      DO 56 I=1,MSTA        8 347
001243      X(I)=X(I)*XCXR        8 348
001245      MAS(I)=MAS(I)*MCMR        8 349
001247      66 CONTINUE          8 350
001252      IT=1                 8 351
001253      IF (IFRQ.EQ.0) GO TO 67 8 352
001254      C      ... USE INPUT FREQUENCIES 8 353
001255      IT=2                 8 354
001256      OMEGA=OMEG(NFRQ)        8 355
001257      67 ITER=1            8 356
001260      NU=1                 8 357
001261      NUCT=0              8 358
001262      GMASS=0.             8 359
001263      C      ... NON-DIMENSIONALIZE FREQUENCY 8 360
001264      C      OMEGA=OMEGA/OMP          8 361
001265      C      ... CALCULATE A MATRICES 8 362
001266      68 CALL AMATRX (1)       8 363
001267      C      ... CREATE U MATRIX 8 364
001268      CALL UMATRX (IFRR)       8 365
001269      IF (INPRNT.NE.0) WRTTE (6,115) OMEGA,UDET(NU) 8 366
001270      C      ... IF IERR = -1, DETERMINANT VALUE IS NO GOOD, GO TO NEXT OMEGA 8 367
001271      C      ... IF IERR = 0, EVERYTHING IS OK 8 368
001272      C      ... IF IERR = 1, 10 BAD DETERMINANTS, GO TO NEXT PROBLEM 8 369
001273      C      ... IF IERR = 68,70,1 8 370
001274      70 GO TO (71,72), IT        8 371
001275      71 CALL ITERAT (IFRR)       8 372
001276      72 IF (IERR.NE.0) GO TO 1        8 373
001277      C      ... IT = 2 IF FREQUENCY ITERATION HAS CONVERGED 8 374
001278      GO TO (68,69), IT        8 375
001279      C      ... RECALCULATE A MATRICES FOR MODAL DATA 8 376
001280      72 CONTINUE          8 377
001281      C      ... RECALCULATE A MATRICES FOR MODAL DATA 8 378
001282      CALL AMATRX (2)       8 379
001283      C      ... DIMENSIONALIZE MODAL DATA 8 380
001284      DO 73 I=1,MSTA        8 381
001285      X(I)=X(I)/XCXR        8 382
001286      MAS(I)=MAS(I)/MCMR        8 383
001287      Y(1,I)=Y(1,I)*RFEA        8 384
001288      Y(2,I)=Y(2,I)*RFX        8 385
001289      DY(1,I)=DY(1,I)*XH
001290

```

001342	73	CONTINUE	B 386
001345		OMEGA=OMEGA*DMR	B 387
001346		GMASS=GMASS*RFMASS*RFX**3	B 388
001351		VALUE=1.	B 389
001353		IF (INORN) 80,80,74	B 390
	C NORMALIZE ON A SPECIFIC STATION (ZETA)	B 391
	C NOPN IS THE DESIRED STATION IN REFERENCE TO THE INPUT. MUST	B 392
	C LOOK FOR THE SAME STATION IN THE EXPANDED ARRAY.	B 393
001354	74	MTOT=0	B 394
001355		LIM=NORMBR-1	B 395
001357		IF (LIM.EQ.0) GO TO 76	B 396
001360		DO 75 M=1,LIM	B 397
001361		MTOT=MTOT+NUM(M)	B 398
001362	75	CONTINUE	B 399
001365		MTOT=MTOT+NORN	B 400
001366		L1=NSTA(LIM)+NORN	B 401
001370		L2=NSTA(NORMBR)	B 402
001372		GO TO 77	B 403
001372	76	L1=NOPN	B 404
001374		L2=NSTA(1)	B 405
001375		MTCT=NORN	B 406
001376	77	DO 78 M=L1,L2	B 407
001400		IF (ABS(STN(MTOT)-X(M)).LE..0001) GO TO 79	B 408
001405	78	CONTINUE	B 409
001410	79	VALUE=Y(2,M)	B 410
001413		GO TO 83	B 411
001413	80	IF (.NOT.NORDIS) GO TO 82	B 412
	C NORMALIZE ON MAXIMUM ZETA	B 413
001415		ALUE=ABS(Y(2,1))	B 414
001417		VALUE=Y(2,1)	B 415
001420		DO 81 I=1,MSTA	B 416
001421		IF (ALUE.GE.ABS(Y(2,I))) GO TO 81	B 417
001425		ALUE=ABS(Y(2,I))	B 418
001430		VALUE=Y(2,I)	B 419
001431	81	CONTINUE	B 420
001434		GO TO 83	B 421
	C NORMALIZE ON GENERALIZED MASS	B 422
001434	82	IF (.NOT.NOREQM) GO TO 83	B 423
001436		VALUE=SQRT(GMASS)	B 424
001440	83	DO 85 I=1,MSTA	B 425
001442		DO 84 K=1,2	B 426
001443		Y(K,I)=Y(K,I)/VALUE	B 427
001446		DY(K,I)=DY(K,I)/VALUE	B 428

```

001451 84 CONTINUF          B 429
001453 85 CONTINUE          B 430
001455      GMASS=GMASS/VALUE**2    B 431
001457      ROMEA=OMEGA/6.2831853   B 432
001461      IF (.NOT.ISOD) GO TO 86   B 433
001461      C .... READ INPUT BACK IN   B 434
001462      REWIND 2                 B 435
001464      READ (2) K,(STN(1),SSAM(1),HI(1),I=1,K)   B 436
001504 86 FORMA(25)=OMFORM(7)       B 437
001506      WRITE (6,119)             B 438
001511      DO 87 I=1,6             B 439
001513      IF (OMEGA.GT.10.**(I-1)) GO TO 87   B 440
001523 FORMA(25)=OMFORM(1)       B 441
001525      GO TO 88               B 442
001525 87 CONTINUF          B 443
001527 88 WRITE (6,FORMA) (TITLE(I),I=1,14),TIME,NFRQ,ROMEA,OMEGA   B 444
001551      IF (.NOT.TORVB) WRITE (6,116) GMASS   B 445
001560      WRITE (6,117) (HEADNG(1,I),HEADNG(2,I),I=1,NVAR)   B 446
001577      WRITE (6,118)             B 447
001603      LINE=9                B 448
001604      N=0                  B 449
001605      NN=0                B 450
001605      I=0                  B 451
001607      NS=1                B 452
001610      DO 97 M=1,NMBR        B 453
001611      LINE=LINE+1           B 454
001613      IF (LINE.LT.LMAX) GO TO 90   B 455
001615      LINE=0              B 456
001616      WRITE (6,119)             B 457
001616      WRITE (6,120) NFRQ,(TITLE(L),L=1,14),(HEADNG(1,II),HEADNG(2,II),II   B 458
001621      I=1,NVAR)            B 459
001650      WRITE (6,118)             B 460
001654 90 N=N+1              B 461
001656      I=I+1              B 462
001657      IF (.NOT.ISOD) GO TO 93   B 463
001660 91 C .... WRITE OUTPUT FOR INPUT STATIONS ONLY   B 464
001660      IF (ABS(STN(NS)-X(I)).LE..0001) GO TO 93   B 465
001666      I=I+1              B 466
001666      GO TO 91              B 467
001667 92 NS=NS+1              B 468
001667      IF (N-NUM(M)) 89,96,96   B 469
001671 93 DO 94 II=1,NVAR        B 470
001674      LL=LV(III)+2*(I-1)     B 471
001676

```

001701 MM=MV(II) 8 472
 001703 SAM(II)=U(LL,MM) 8 473
 001707 94 CONTINUE 8 474
 001711 WRITE (6,FORM) M,N,X(I),(SAM(II),II=1,NVAR) 8 475
 001732 IF (ISOD) GO TO 92 8 476
 001734 IF (N-NSTA(M)+NN) 89,95,95 8 477
 001737 95 NN=NSTA(M) 8 478
 001741 96 N=0 8 479
 001742 97 CONTINUE 8 480
 001745 IF (IFRO.EQ.0) OMEGA=OMEGA+DELOMG*OMR 8 481
 001751 IF (PLOT) 98,99 8 482
 001753 98 ICOUNT=1COUNT+1 8 483
 001755 IF (ICOUNT.GT.3) GO TO 99 8 484
 001760 CALL STORPLT (NSUM,NUM,ISOD,TORVB,IXPL) 8 485
 001763 99 CONTINUE 8 486
 001766 IF (PLOT.AND.ICOUNT.GE.3) RETURN 8 487
 001775 GO TO 1 8 488
 C 8 489
 001776 100 FORMAT (A2,13A6) 8 490
 001776 101 FORMAT (1H19X,A2,13A6) 8 491
 001776 102 FORMAT (1H09X,A2,13A6) 8 492
 001776 103 FORMAT (5X,A2,A6,5X,42H---THIS CARD IS IN ERROR. JOB TERMINATED.) 8 493
 001776 104 FORMAT (5X,42HMORE THAN 600 STATIONS. RUN TERMINATED.) 8 494
 001776 105 FORMAT (1H15X,20HPARAMETER CONTROLS -A2,13A6//) 8 495
 001776 106 FORMAT (19X,A6,OPE13.5,12X,8HNORMODE =I4//17X,A8,E13.5,12X,8HNORMBR 8 496
 1 =I4//20X,SHRFX =E13.5,14X,8HNOPN =I4//13X,12HCMEGA SUBR =E13.5,12 8 497
 2X,8HNORDIS = A2//19X,4HDELX =E13.5,12X,A9,A3//13X,7HCMEGA =E13.5/ 8 498
 3/17X,A8,E13.5,12X,8HDELOMG =E13.5/18X,A7,E13.5,12X,8HOMGTOL =E13. 8 499
 45//) 8 500
 001776 107 FORMAT (1H05X,19HBOUNDARY CONDITIONS//6X,5HJOINT4X,53HMBRI MBRJ 8 501
 1 MBRK MBRL T0JTRN T1JTRN T1KTRN T1LTRN//(6X,I3,2X,817//) 8 502
 001776 108 FORMAT (1H15X,32HSPTNG AND FLEXIBILITY CONSTANTS//6X,5HJOINT4X,63 8 503
 1HMBRT MBRJ MBRL TRNS TRNFIJ TRNFIK TRNFIL// 8 504
 215X,I3,2X,417,0P4E10.3)) 8 505
 001776 109 FORMAT (1H 50X,7HTABLE I//43X,24HPHYSICAL CHARACTERISTICS/43X,3A10 8 506
 1,//36X,A2,13A6//7X,7HSTATION7X,1HX12X,A4,13X,A2// 8 507
 001776 110 FORMAT (1H 40X,19HTABLE I (CONTINUED)//7X,7HSTATION7X,1HX12X,A4,14- 8 508
 1X,A2//) 8 509
 001776 111 FORMAT (1B,1H-I3,0PF12.3,0P2E16.5) 8 510
 001776 112 FORMAT (1B,1H-I3,0PF12.3,0P2E16.5) 8 511
 001776 113 FORMAT (//20X,22HCENTER OF GRAVITY X = F10.5//20X,22HTOTAL MASS 8 512
 1 = F10.5//20X,22HS = OPE14.5) 8 513
 001776 114 FORMAT (1H1) 8 514

001776	115	FORMAT (5X,16HNON-DIM. FREQ = 1PE16.7,5X,8HDETERM =E16.7)	8 515
001776	116	FORMAT (4X,17H GENERALIZED MASS,E16.7)	8 516
001776	117	FORMAT (/5X,7HSTATION6X,1HX,2X,4(2X,2A6))	8 517
001776	118	FORMAT (//)	8 518
001776	119	FORMAT (1H// ////)	8 519
001776	120	FORMAT (1H 42X,14HTABLE II, MODEI3//36X,A2,15A6//5X,7HSTATION6X,1 1HX,2X,4(2X,2A6))	8 520
001776		END	8 521
001776			8 522-

E6

```

SUBROUTINE STOPPLT (NSUM,NUM,ISOD,TORVB,IXPL)      C 1
DIMENSTON NUM(30), Y(2,600), DY(2,600), STN(600), SSAM(600), NS(30) C 2
1), HI(600)                                         C 3
000010 LOGICAL TORVB,ISOD                           C 4
000010 REAL MAS                                     C 5
000010 COMMON /BLK2/ X(600),HH(600),MAS(600),NSTA(30) C 6
000010 COMMON /BLK3/ U(100,60),B(2,2,30),OMEGA,NOMBR,TEMP(2,30),GMASS C 7
000010 EQUIVALENCE (U,Y), (U(1,13),DY), (U(1,25),STN), (U(1,31),SSAM), (U C 8
000010 1(1,37),HI)                                 C 9
000010 IF (ISOD) GO TO 5                            C 10
000011 NS(1)=NSTA(1)                                C 11
000012 NI=NOMBR-1                                  C 12
000014 DO 1 I=1,NI                                 C 13
000016 1 NS(I+1)=NSTA(I+1)-NSTA(I)                C 14
000023 NSUM=NSTA(NOMBR)                            C 15
000024 WRITE (6,11) NSUM                           C 16
000032 IF (IXPL.EQ.0) 2,3                         C 17
000041 2 CALL RECOUT (3,1,0,TORVB,NOMBR)          C 18
000044 CALL RECOUT (3,2,0,NS,1,NOMBR,1)            C 19
000053 IXPL=1                                      C 20
000057 CALL RECOUT (3,1,0,NSUM)                    C 21
000063 WRITE (4) (X(I),I=1,NSUM),(MAS(I),I=1,NSUM),(HH(I),I=1,NSUM) C 22
000132 3 DO 4 K=1,NSUM                            C 23
000137 CALL PECOUT (3,1,0,Y(2,K),DY(2,K),Y(1,K)) C 24
000152 4 CONTINUE.                                C 25
000160 RETURN.                                     C 26
000161 C PLOT DATA FOR INPUT STATIONS ONLY        C 27
000162 NSUM=0                                       C 28
000163 DO 5 I=1,NOMBR                            C 29
000164 NSUM=NSUM+NUM(I)                          C 30
000167 WRITE (6,11) NSUM                         C 31
000174 IF (IXPL.EQ.0) 7,8                         C 32
000203 7 IXPL=1                                  C 33
000204 CALL RECOUT (3,1,0,TORVB,NOMBR)          C 34
000207 CALL RECOUT (3,2,0,NUM,1,NOMBR,1)         C 35
000221 CALL RECOUT (3,1,0,NSUM)                  C 36
000230 WRITE (4) (STN(I),I=1,NSUM),(SSAM(I),I=1,NSUM),(HI(I),I=1,NSUM) C 37
000277 8 NS=0                                     C 38
000300 I=1                                       C 39
000301 DO 10 J=1,NOMBR                         C 40
000306 NU=NUM(J)                                C 41
000310 DO 10 II=1,NU                           C 42

```

000311	NS=NS+1	C 43
000313	9 IF (ABS(STN(NS)-X(I)).LE..0001) GO TO 10	C 44
000321	I=I+1	C 45
000322	GO TO 9	C 46
000322	10 CALL RECOUNT (3,1,0,Y(2,I),DY(2,I),Y(1,I))	C 47
000345	RETURN	C 48
	C	C 49
000346	11 FORMAT (/21H TOTAL NO. STATIONS =,15)	C 50
000346	END	C 51-

SUBROUTINE STORE (A,N,M,I)
DIMENSION A(1,2)
COMMON /BLK3/ U(100,60),B(2,2,30),OMEGA,NOMBR,TEMP(2,30),GMASS
JJ=M
DO 4 J=1,2
IF (I) 1,2,2
1 U(N,JJ)=-A(1,J)
GO TO 3
2 U(N,JJ)=A(1,J)
3 JJ=JJ+1
4 CONTINUE
RETURN
END

0 1
0 2
0 3
0 4
0 5
0 6
0 7
0 8
0 9
0 10
0 11
0 12
0 13-

```

SUBROUTINE CGRAV (NOMBR,TOTM,S,CG)
COMMON /BLK2/ X(600),HH(600),MAS(600),NSTA(30)
REAL MAS
SUM1=0.
SUM2=0.
L1=?
L2=NSTA(1)
DO 2 M=1,NOMBR
DO 1 I=L1,L2
SUM1=SUM1+.5*(MAS(I)+MAS(I-1))*(X(I)-X(I-1))
SUM2=SUM2+.25*(MAS(I)+MAS(I-1))*(X(I)**2-X(I-1)**2)
CONTINUE
L1=NSTA(M)+2
L2=NSTA(M+1)
CONTINUE
TOTM=SUM1
S=SUM2
CG=S/TOTM
RETURN
END

```

	I
E	2
E	3
E	4
E	5
E	6
E	7
E	8
E	9
E	10
E	11
E	12
E	13
E	14
E	15
E	16
E	17
E	18
E	19
E	20

000006 SUBROUTINE MOVE (A,C,N)
000006 DIMENSION C(N,2), A(N,2)
000007 DO 2 I=1,N
000007 DO 1 J=1,2
000010 C(I,J)=A(I,J)
000021 1 CONTINUE
000022 2 CONTINUE
000024 RETURN
000025 END

1
2
3
4
5
6
7
8
9

000010	SUBROUTINE MATMPY (A,B,C,N,M)	G 1
C	DIMENSTON A(N,2), B(2,M), C(N,M)	G 2
C N = NO. OF ROWS IN A AND C	G 3
 M = NO. OF COLUMNS IN B AND C	G 4
000010	DO 2 I=1,N	G 5
000011	DO 1 J=1,M	G 6
000012	C(I,J)=A(I,1)*B(1,J)+A(I,2)*B(2,J)	G 7
000043 1	CONTINUE	G 8
000046 2	CONTINUE	G 9
000050	RETURN	G 10
000051	END	G 11-

```

SUBROUTINE INTERP (XX,YY,I)
DIMENSION X(4), Y(4), XX(20), YY(20) H 1
000006 IF (I.LT.3) GO TO 6 H 2
000006 X(1)=XX(I-2) H 3
000010 X(2)=XX(I-1) H 4
000011 X(3)=XX(I) H 5
000013 X(4)=YY(I-2) H 6
000014 Y(1)=YY(I-1) H 7
000016 Y(2)=YY(I) H 8
000017 Y(3)=YY(I) H 9
000021 S1=(X(1)-X(2))/(Y(1)-Y(2)) H 10
000025 S2=(X(2)-X(3))/(Y(2)-Y(3)) H 11
000030 IF (ABS(S1/S2-1.).LE.1.E-6) GO TO 5 H 12
000035 XMAX=MAX(X(1),X(2),X(3))+1.E-7 H 13
000044 XMIN=MIN(X(1),X(2),X(3))-1.E-7 H 14
000053 DENOM=(X(1)**2-X(3)**2)*(X(2)-X(3))-(X(2)**2-X(3)**2)*(X(1)-X(3)) H 15
000065 A=((Y(1)-Y(3))*(X(2)-X(3))-(Y(2)-Y(3))*(X(1)-X(3)))/DENOM H 16
000077 B=((X(1)**2-X(3)**2)*(Y(2)-Y(3))-(X(2)**2-X(3)**2)*(Y(1)-Y(3)))/DE H 17
1 NOW
000112 C=Y(3)-A*X(3)**2-B*X(3) H 18
000120 IF (A.LT.1.E18.AND.B.LT.1.E18.AND.C.LT.1.E18) GO TO 1 H 19
000133 B=B/1.E18 H 20
000134 B=B/1.E18 H 21
000135 C=C/1.E18 H 22
000136 1 SQP=B**2-4.*A*C H 23
000142 IF (SQP) 7,3,2 H 24
000144 2 SQP=SQRT(SQP) H 25
000147 3 R1=(-B+SQP)/(2.*A) H 26
000153 R2=(-B-SQP)/(2.*A) H 27
000157 XX(I+1)=0. H 28
000162 IF (R1.GT.XMIN.AND.R1.LT.XMAX) XX(I+1)=R1 H 29
000174 IF (R2.GT.XMIN.AND.R2.LT.XMAX) XX(I+1)=R2 H 30
000206 IF (XX(I+1).EQ.0.) GO TO 7 H 31
000210 4 CONTINUE H 32
000210 RETURN H 33
000211 5 XX(I+1)=X(2)-Y(2)*S2 H 34
000215 GO TO 4 H 35
000216 6 XX(3)=XX(1)-YY(1)*(XX(1)-XX(2))/(YY(1)-YY(2)) H 36
000225 GO TO 4 H 37
000225 7 J=I H 38
000226 8 J=J-1 H 39
000230 IF (YY(J)*YY(I)) 9,9,8 H 40
000233 9 XX(I+1)=XX(J)-YY(J)*(XX(J)-XX(I))/(YY(J)-YY(I)) H 41
000233 H 42

```

000244
000245

GO TO 4
END

H 43
H 44-

```

10
      SUBROUTINE ITRAT (IERR)
      DIMENSION CM(20)
      COMMON /BLK3/ U(100,60),B(2,2,30),OMEGA,NOMBR,TEMP(2,30),GMASS
      COMMON /BLK4/ UDET(20),NU,DELOMG,OMGTOL,IT,ITER,NUCT
      IERR=0
      GO TO 1,5, ITER
      1 IF (NU.NE.1) GO TO 2
      2 OM(1)=OMEGA
      3 GO TO 3
      C .... TEST FOR SIGN CHANGE
      4 IF (UDET(NU-1)*UDET(NU)) 4,9,3
      C .... INCREMENT FREQUENCY AND TRY AGAIN
      5 NU=NU+1
      6 IF (NU.GT.20) GO TO 14
      7 OM(NU)=OM(NU-1)+DELOMG
      8 GO TO 7
      C .... STGN HAS CHANGED. BEGIN ITERATION.
      9 ITER=2
      10 IF (NU.LE.?) GO TO 6
      11 UDET(1)=UDET(NU-2)
      12 UDET(2)=UDET(NU-1)
      13 UDET(?)=UDET(NU)
      14 OM(1)=OM(NU-2)
      15 OM(2)=OM(NU-1)
      16 OM(?)=OM(NU)
      17 NU=?
      18 GO TO A
      C .... TEST FOR CONVERGENCE
      C .... IS DETERMINANT NEARLY ZERO
      19 IF (ABS(UDET(NU)).LE.1.E-6) GO TO 9
      C .... IS CHANGE IN DETERMINANT NEARLY ZERO
      20 IF (ABS(UDET(NU)/UDET(NU-1)-1.).LE.1.E-6) GO TO 9
      C .... IS CHANGE IN FREQUENCY LESS THAN TOLERANCE
      21 IF (ABS(OM(NU)/OM(NU-1)-1.).LE.OMGTOL) GO TO 9
      22 IF (NU.EQ.20) GO TO 10
      23 CALL INTERP (CM,UDET,NU)
      24 NU=NU+1
      25 IT=1
      26 OMFGA=OM(NU)
      27 RETURN
      C .... GOOD FREQUENCY. RETURN TO CALCULATE MODAL DATA.
      28 IT=2
      29
      30
      31
      32
      33
      34
      35
      36
      37
      38
      39
      40
      41
      42

```

```

      SUBROUTINE UMATRIX (IERP)
      DIMENSION C(1,2), D(1,2), E(1,2), F(1,2), G(1,2), H(1,2), P(1,2),
     1Q(1,2), R(1,2), S(1,2), BASE1(1,2), BASE2(1,2), ERASE(100)
      DIMENSION COL(120)
      COMMON /BLK1/ JC(30,8),TRNS(30),TRNFIJ(30),TRNFIK(30),TRNFIL(30),J
      INT
      COMMON /BLK2/ U(100,60),B(2,2,30),OMEGA,NOMBR,TEMP(2,30),GMASS
      COMMON /BLK4/ UDET(20),NU,DELOMG,OMGTOL,IT,ITER,NUCT
      EQUIVALENCE (C,D,E,F), (G,P,R), (H,Q,S), (COL,TEMP)
      DATA BASE1/1..0./,BASE2/0..1./
      IERR=0
      LIM=2*NOMBR
      DO 2 J=1,LIM
      DO 1 I=1,LIM
      U(I,J)=0.
      1 CONTINUE
      2 CONTINUE
      JP1=1
      JR2=2
      DO 21 M=1,JNT
      C .... IS THERE AN -I- BEAM
      ITPIG=1
      IF (JC(M,1)) 3,27,3
      3 CALL MOVE (BASE1,C,1)
      KK=2*(JC(M,1)-1)+1
      LL=JP1
      IF (JC(M,5)=2) 4,5,6
      4 C(1,1)=0.
      C(1,2)=1.
      ITRIG=2
      GO TO 6
      5 C(1,2)=TRNS(M)
      N=JC(M,1)
      CALL MATMPY (C,B(1,1,N),TEMP,1,2)
      CALL STORE (TEMP,LL,KK,1)
      C .... IS THERE A -J- BEAM
      6 IF (JC(M,2)) 7,13,7
      7 CALL MOVE (BASE1,D,1)
      CALL MOVE (BASE2,G,1)
      CALL MOVE (BASE2,H,1)
      GO TO (9,8), ITRIG
      8 D(1,1)=0.

```

EOI

000261	CALL MOVE (BASE2,S,1)	J 86
000264	GO TO (22,22), ITRTG	J 87
000273	22 F(1,1)=0.	J 88
000274	23 KKK=2*(JC(M,4)-1)+1	J 89
000277	CALL STORE (F,LL,KKK,-1)	J 90
000302	IF (JC(M,8)-2) 26,24,25	J 91
000307	24 S(1,1)=-TRNFI(M)	J 92
000311	GO TO 26	J 93
000312	25 R(1,2)=0.	J 94
000313	S(1,2)=0.	J 95
000314	S(1,1)=1.	J 96
000315	26 LLL=JR2	J 97
000317	JR2=JR2+1	J 98
000320	N=JC(M,1)	J 99
000322	CALL MATMPY (P,B(1,1,N),TEMP,1,2)	J 100
000327	CALL STORE (TEMP,LLL,KK,1)	J 101
000332	CALL STORE (S,LLL,KKK,-1)	J 102
000335	GO TO 21	J 103
000337	27 CALL MOVE (BASE1,D,1)	J 104
000342	KK=2*(JC(M,2)-1)+1	J 105
000345	LL=J91	J 106
000347	IF (JC(M,5)-2) 28,29,30	J 107
000352	28 D(1,1)=0.	J 108
000353	D(1,2)=-1.	J 109
000355	GO TO 30	J 110
000355	29 D(1,2)=-TRNS(M)	J 111
000357	30 CALL STORE (D,LL,KK,-1)	J 112
000362	31 J*1=J*2	J 113
000364	JR2=JP2+1	J 114
000365	32 CONTINUF	J 115
000371	IF (IT.EQ.2) GO TO 37	J 116
000373	ISCALE=0	J 117
000374	CALL SIMFO (U,LTM,0,0,UDET(NU),TEMP,100,ISCALE)	J 118
000404	IF (ISCALE.EQ.0) GO TO 35	J 119
000406	CALL OVERFL (J)	J 120
000410	DFT=UDET(NU)*1.E18**ISCALE	J 121
000416	CALL OVERFL (J)	J 122
000417	GO TO (33,34), J	J 123
000426	33 ISCALE=18*ISCALE	J 124
000430	WRITE (6,*0) OMEGA,UDET(NU),ISCALE	J 125
000442	IERR=-1	J 126
000444	NU=1	J 127
000445	LARGE=LARGE+1	J 128

```

000104   9      KKK=2*(JC(M,2)-1)+1
000107
000112   JF (JC(M,6)-2) 12,10,11
000117   10      H(1,1)=-TRNFIJ(M)
000121
000122   11      G(1,2)=0.
000123
000124   H(1,2)=0.
000125   12      H(1,1)=1.
000127
000128   LLL=JR2
000129   JR2=JR2+1
000130   CALL STORE (H,LLL,KKK,-1)
000133   N=JC(M,1)
000135   CALL MATMPY (G,B(1,1,N),TEMP,1,2)
000142   CALL STORE (TEMP,LLL,KK,1)
C     .... IS THERE A -K- BEAM
000145   13      IF (JC(M,3)) 14,20,14
000150   14      CALL MOVE (BASE1,F,1)
000153   CALL MOVE (BASE2,P,1)
000156   CALL MOVE (BASF2,Q,1)
000161   GO TO (16,15), ITRTG
000170   15      E(1,1)=0.
000171   16      KKK=2*(JC(M,3)-1)+1
000174   N=JC(M,3)
000176   CALL MATMPY (E,B(1,1,N),TEMP,1,2)
000203   CALL STORE (TEMP,LL,KKK,1)
000206   IF (JC(M,7)-2) 19,17,18
000213   17      Q(1,1)=TRNFTK(M)
000215
000216   18      GO TO 19
000217   P(1,2)=0.
000218   Q(1,2)=0.
000220   Q(1,1)=1.
000221   19      LLL=JR2
000223   JR2=JR2+1
000224   N=JC(M,1)
000226   CALL MATMPY (P,B(1,1,N),TEMP,1,2)
000233   CALL STORE (TEMP,LLL,KK,1)
000236   N=JC(M,2)
000240   CALL MATMPY (Q,B(1,1,N),TEMP,1,2)
000245   CALL STORE (TEMP,LLL,KKK,-1)
C     .... IS THERE AN -L- BEAM
000250   20      IF (JC(M,4)) 21,31,21
000253   21      CALL MOVE (BASE1,F,1)
000256   CALL MOVE (BASE2,R,1)

```

SOT

```

000447      OMEGA=OMEGA+DELOMG          J 129
000451      IF (LARGE.GT.10) IERR=1     J 130
000454      GO TO 26                 J 131
000455 34      UDET(NU)=DET         J 132
000457 35      LARGE=0              J 133
000460 36      RETURN               J 134
000461 37      CONTINUE             J 135
C      ---- CHECK FOR NON-ZERO ELEMENT IN FIRST BEAM
000461      DO 38 M=1,JNT            J 136
000463      IF (JC(M,2).EQ.1) GO TO 39  J 137
000465 38      CONTINUE             J 138
000467 39      TF (JC(M,5)-2) 40,41,41 J 139
000472 40      II=1                J 140
000473      GO TO 42                J 141
000474 41      II=2                J 142
000475 42      CONTINUE             J 143
000475      DO 43 J=1,LIM           J 144
000477      COL(J)=-U(J,II)        J 145
000504 43      CONTINUE             J 146
C      ---- REMOVE SELECTED ROW AND COLUMN
000507      LLTM=II+1              J 147
000511      DO 45 J=LLIM,LIM        J 148
000512      DO 44 T=1,LIM           J 149
000513      U(I,J-1)=U(I,J)        J 150
000522 44      CONTINUE             J 151
000524 45      CONTINUE             J 152
000526      N=LIM-1              J 153
000520      DO 47 T=LLIM,LIM        J 154
000532      DO 46 J=1,N             J 155
000533      U(I-1,J)=U(I,J)        J 156
000542 46      CONTINUE             J 157
000544      COL(I-1)=COL(I)        J 158
000546 47      CONTINUE             J 159
C      ---- SOLVE FOR MODAL MATRIX
000551      CALL STMEQ (U,LIM-1,COL,1,DET,ERASE,100,ISCALE) J 160
C      ---- INSERT 1. WHERE ROW AND COLUMN REMOVED
000562      I=LIM                J 161
000564 48      IF (I.LE.III) GO TO 49  J 162
000570      COL(I)=COL(I-1)        J 163
000571      I=I-1                J 164
000573 49      GO TO 48              J 165
000573      COL(III)=1.          J 166
000575      GO TO 36              J 167
                                J 168
                                J 169
                                J 170
                                J 171

```

000576 C
000576 50 FORMAT (//5X,16HNON-DIM. FREQ = 1PE16.7,5X,8HDETERM =E16.7,7H * 10 J 172
1**I3) J 173
000576 END J 174
J 175-

109

```

SUBROUTINE AMATRX (IX)
DIMENSION A(2,2), W(2,2), Y(2,600), DY(2,600)
DIMENSTON EA(600)
COMMON /BLK2/ X(600),HH(600),MAS(600),NSTA(30)
COMMON /BLK3/ U(100,60),B(2,2,30),OMEGA,NOMBR,TEMP(2,30),GMASS
EQUIVALENCE (U,Y), (U(1,13),DY), (HH,EA)
REAL MAS
FRC=OMEGA
FRQ2=FRQ**2
L=-1
DO 8 M=1,NOMBR
K=L+2
L=NSTA(M)-1
DO 7 I=K,L
DX=X(I+1)-X(I)
IF (DX.EQ.0.) GO TO (7,3), IX
DX2=DX*DX
C .... CALCULATE A MATRIX
A(1,1)=1.-(FRQ2*MAS(I+1)*DX2)/(2.*EA(I))
A(1,2)=- (MAS(I+1)+MAS(I))*DX*FRQ2/2.
A(2,1)=DX/2.*(1./EA(I+1)+1./EA(I))
A(2,2)=1.-MAS(I)*FRQ2*DX2/(2.*EA(I+1))
GO TO (5,1), IX
C .... CALCULATE MODAL DATA FOR EACH STATION
1 IF (I.GT.K) GO TO 2
C .... MOVE TEMP TO Y FOR FIRST STATION ON BEAM
Y(1,I)=TEMP(1,M)
Y(2,I)=TEMP(2,M)
W(1,1)=W(2,1)=0.
W(1,2)=-MAS(I)*FRQ2
W(2,1)=1./EA(I)
C .... MULTIPLY W X Y TO FIND DY FOR FIRST STATION ON BEAM
000106 CALL MATMPY (W(1,1),Y(1,I),DY(1,I),2,1)
C .... MULTIPLY A(I) X Y(I) TO FIND Y(I+1)
000114 2 CALL MATMPY (A(1,I),Y(1,I),Y(1,I+1),2,1)
W(1,1)=W(2,2)=0.
W(1,2)=-MAS(I+1)*FRQ2
W(2,1)=1./EA(I+1)
C .... MULTIPLY W(I+1) X Y(I+1) TO FIND DY(I+1)
000132 CALL MATMPY (W(1,I),Y(1,I+1),DY(1,I+1),2,1)
000140 GMASS=GMASS+(1./6.)*DX*(MAS(I)*(2.*Y(2,I)**2+Y(2,I+1)**2)+MAS(I+1)
1*(Y(2,I)**2+2.*Y(2,I+1)**2))

```

K	1
K	2
K	3
K	4
K	5
K	6
K	7
K	8
K	9
K	10
K	11
K	12
K	13
K	14
K	15
K	16
K	17
K	18
K	19
K	20
K	21
K	22
K	23
K	24
K	25
K	26
K	27
K	28
K	29
K	30
K	31
K	32
K	33
K	34
K	35
K	36
K	37
K	38
K	39
K	40
K	41
K	42

```

000101.    ICK=0          I  43
000102.    GO TO 7        I  44
000104. 10  CONTINUE      I  45
000104.    IF (ICK.EQ.1) GO TO 13   I  46
000106.    C    .... TRY ANOTHER 20 ITERATIONS AVOIDING BAD SPOT IF POSSIBLE I  47
000107.    ICK=1          I  48
000107.    III=2          I  49
000110. 11  IF (UDET(III)*UDET(20).LT.0.) GO TO 12   I  50
000113.    III=III-1       I  51
000114.    GO TO 11        I  52
000115. 12  UDET(2)=UDET(III)      I  53
000117.    UDET(1)=UDET(20)      I  54
000121.    OM(2)=OM(III)      I  55
000122.    OM(1)=OM(20)        I  56
000124.    OM(3)=-(OM(1)-OM(2))/2.+OM(1)    I  57
000127.    NU=3            I  58
000130.    GO TO 7          I  59
000131. 13  WRITE (6,16)      I  60
000135.    WRITE (6,18) (OM(I),UDET(I),I=1,NU)    I  61
000152.    IERR=1          I  62
000154.    GO TO 8          I  63
000155. 14  NUCT=NUCT+1      I  64
000157.    IF (NUCT.EQ.10) GO TO 15    I  65
000161.    UDET(1)=UDET(NU-2)      I  66
000163.    UDET(2)=UDET(NU-1)      I  67
000164.    OM(1)=OM(NU-2)        I  68
000166.    OM(2)=OM(NU-1)        I  69
000167.    OM(2)=OM(2)+DELOMG  I  70
000171.    NU=3            I  71
000172.    GO TO 7          I  72
000172. 15  WRITE (6,17)      I  73
000176.    NU=NU-1          I  74
000200.    WRITE (6,18) (OM(I),UDET(I),I=1,NU)    I  75
000214.    IERR=1          I  76
000216.    GO TO 8          I  77
000217.    C
000217. 16  FORMAT (5X,35HNO CONVERGENCE AFTER 20 ITERATIONS.) I  78
000217. 17  FORMAT (5X,80H200 FREQUENCY CHANGES WITH NO SIGN CHANGE IN THE U D I  79
1ETERMINANT. RUN TERMINATED.) I  80
000217. 18  FORMAT (/5X,16HN0N-DIM. FREQ = OPE16.7,5X,8MDETERM =E16.7)) I  81
000217.    END             I  82
                                         I  83-

```

000162	GO TO 7	K 43
	C REPEAT VALUES FOR DUPLICATE STATIONS	K 44
000164	DO 4 II=1,2	K 45
000166	Y(II,I+1)=Y(II,I)	K 46
000172	DY(II,I+1)=DY(II,I)	K 47
000176	4 CONTINUE	K 48
000177	GO TO 7	K 49
	C CALCULATE B MATRICES	K 50
000200	5 IF (I.GT.K) GO TO 6	K 51
	C MOVE A TO B FOR FIRST STATION ON BEAM	K 52
000204	CALL MOVE (A(1,1),B(1,1,M),2)	K 53
000210	GO TO 7	K 54
	C MULTIPLY A X B, STORE IN TEMP	K 55
000212	6 CALL MATMPY (A(1,1),B(1,1,M),TEMP(1,1),2,2)	K 56
	C MOVE TEMP TO B	K 57
000220	CALL MOVE (TEMP(1,1),B(1,1,M),2)	K 58
000225	7 CONTINUE	K 59
000231	8 CONTINUE	K 60
000233	RETURN	K 61
000234	END	K 62-

OTT

```

      SUBROUTINE SIMEQ (A,N,B,M,DETERM,IPIVOT,NMAX,ISCALE)
C   SOLUTION OF SIMULTANEOUS LINEAR EQUATIONS
C
000013 C   DIMENSION IPIVOT(N), A(NMAX,N), B(NMAX,M)
000013 C   EQUIVALENCE (IROW,JROW), (ICOLUMN,JCOLUMN), (AMAX,T,SWAP)
000013 C   DATA R1,R2/1.E18,1.E-18/
C
C   INITIALIZATION
C
000013 1  ISCALE=0
000014 DETERM=1.0
000015 DO 2 J=1,N
000017 2  IPIVOT(J)=0
000022 DO 37 I=1,N
C
C   SEARCH FOR PIVOT ELEMENT
C
000024 AMAX=0.0
000025 DO 7 J=1,N
000027 3  IF (IPIVOT(J)-1) 3,7,3
000032 3  DO 6 K=1,N
000024 4  IF (IPIVOT(K)-1) 4,6,38
000037 4  IF (ABS(AMAX)-ABS(A(J,K))) 5,6,6
000051 5  IROW=J
000053 ICOLUMN=K
000054 AMAX=A(J,K)
000061 6  CONTINUE
000064 7  CONTINUE
000067 8  IF (AMAX) 9,8,9
000070 8  DETERM=0.0
000071 9  ISCALE=0
000072 GO TO 38
000073 9  IPIVOT(ICOLUMN)=IPIVOT(ICOLUMN)+1
C
C   INTERCHANGE ROWS TO PUT PIVOT ELEMENT ON DIAGONAL
C
000076 10 IF (IROW-ICOLUMN) 10,14,10
000100 10 DETERM=-DETERM
000101 10 DO 11 L=1,N
000103 10 SWAP=A(IROW,L)
000110 10 A(IROW,L)=A(ICOLUMN,L)
000120 11 A(ICOLUMN,L)=SWAP

```

C	1
C	2
C	3
C	4
C	5
C	6
C	7
C	8
C	9
C	10
C	11
C	12
C	13
C	14
C	15
C	16
C	17
C	18
C	19
C	20
C	21
C	22
C	23
C	24
C	25
C	26
C	27
C	28
C	29
C	30
C	31
C	32
C	33
C	34
C	35
C	36
C	37
C	38
C	39
C	40
C	41
C	42

000124	IF (M) 14,14,12	L 43
000125 12	DO 13 L=1,M	L 44
000127	SWAP=B(IROW,L)	L 45
000134	B(IROW,L)=B(ICOLUMN,L)	L 46
000144 13	B(ICOLUMN,L)=SWAP	L 47
000150 14	PIVOT=A(ICOLUMN,ICOLUMN)	L 48
C		L 49
C SCALE THE DETERMINANT		L 50
C		L 51
000155	PIVOTI=PIVOT	L 52
000156	IF (ABS(DETERM)-R1) 17,19,19	L 53
000161 15	DETERM=DETERM/R1	L 54
000163	ISCALE=ISCALE+1	L 55
000164	IF (ABS(DETERM)-R1) 20,16,16	L 56
000167 16	DETERM=DETERM/R1	L 57
000171	ISCALE=ISCALE+1	L 58
000172	GO TO 20	L 59
000173 17	IF (ABS(DETERM)-R2) 18,18,20	L 60
000176 18	DETERM=DETERM*R1	L 61
000200	ISCALE=ISCALE-1	L 62
000201	IF (ABS(DETERM)-R2) 19,19,20	L 63
000204 19	DETERM=DETERM*R1	L 64
000206	ISCALE=ISCALE-1	L 65
000207 20	IF (ABS(PIVOTI)-R1) 23,21,21	L 66
000213 21	PIVOTI=PIVOTI/R1	L 67
000215	ISCALE=ISCALE+1	L 68
000217	IF (ABS(PIVOTI)-R1) 26,22,22	L 69
000221 22	PIVOTI=PIVOTI/R1	L 70
000223	ISCALE=ISCALE+1	L 71
000225	GO TO 24	L 72
000225 23	IF (ABS(PIVOTI)-R2) 24,24,26	L 73
000231 24	PIVOTI=PIVOTI*R1	L 74
000233	ISCALE=ISCALE-1	L 75
000235	IF (ABS(PIVOTI)-R2) 25,25,26	L 76
000240 25	PIVOTI=PIVOTI*R1	L 77
000242	ISCALE=ISCALE-1.	L 78
000244 26	DETERM=DETERM*PIVOTI	L 79
C		L 80
C DIVIDE PIVOT ROW BY PIVOT ELEMENT		L 81
C		L 82
000246	DO 28 L=1,N	L 83
000247	IF (IPIVOT(L)-1) 27,28,38	L 84
000252 27	A(ICOLUMN,L)=A(ICOLUMN,L)/PIVOT	L 85

000260	28	CONTINUE	L 86
000263		IF (M) 31,31,29	L 87
000264	29	DO 30 L=1,M	L 88
000266	30	B(ICOLUMN,L)=B(ICOLUMN,L)/PIVOT	L 89
	C		L 90
	C	REDUCE NON-PIVOT ROWS	L 91
	C		L 92
000276	31	DO 37 L1=1,N	L 93
000300		IF (L1-ICOLUMN) 32,37,32	L 94
000302	32	T=A(L1,ICOLUMN)	L 95
000310		DO 34 L=1,N	L 96
000311		IF (PIVOT(L)-1) 33,34,38	L 97
000314	33	A(L1,L)=A(L1,L)-A(ICOLUMN,L)*T	L 98
000325	34	CONTINUE	L 99
000330		IF (M) 37,37,35	L 100
000331	35	DO 36 L=1,M	L 101
000333	36	B(L1,L)=B(L1,L)-B(ICOLUMN,L)*T	L 102
000351	37	CONTINUE	L 103
000356	38	RETURN	L 104
000357		END	L 105-
112	*FOLLOWING VARIABLES EQUIVALENCED BUT NOT REFERENCED		
	JROW		
	JCOLUMN		

```

OVEPLAY(LINK,2,0)
PROGRAM PLOT
COMMON NPLOT
INTEGER PLTZ,PLTZPR,PLTT
INTEGER PLTM,PLTZR,PLTAE,PLTJG
LOGICAL TORVB
REAL MAS,MZRMN
C
EQUIVALENCE (ZPMIN(1),ZPPIN1), (ZPMIN(2),ZPMIN2), (ZPMIN(3),ZPMIN3)
1, (DZP(1),DZP1), (DZP(2),DZP2), (DZP(3),DZP3)
C
DIMENSION NS(12), X(600), MAS(600), Y(600), Y1(600,3), DY(600,3),
IXP(602), YP(1802), ZPMIN(3), DZP(3), HEADER(8)
C
NAMELIST /NAM1/ INCHX,DX,XMIN,PLTZ,PLTZPR,NZAPR,MPLT,PLTT,NPLTT,TH
1IN,ZPMIN1),DZP1,ZPMIN2,DZP2,ZPMIN3,DZP3,OTT,PLTM,NMZR,MZRMN,DMZR,P
2LTZP,PLTAE,PLTJG,NMAEJG,AJMIN,DAEJG
C
REWIND 3
REWIND 4
PLTZ=PLTZPP=PLTT=MPLT=0
PLTM=PLTZR=PLTAE=PLTJG=0
NZAPR=NPLTT=1
NMZR=NMAEJG=1
READ (5,28) HEADER
READ (5,NAM1)
WRITE (6,NAM1)
CALL NOTATE (0.,0.,.14,HEADER,90.,80)
CALL CALPLT (5.,0.,-3)
CALL RECIN (3,1,2,TORVB,NOMBR)
CALL RECIN (3,2,NOMAR,NS,1,NOMBR,1)
CALL RFCTN (3,1,1,NSUM)
READ (4) (X(I),I=1,NSUM),(MAS(I),I=1,NSUM),(Y(I),I=1,NSUM)
XINCH=INCHX
IF (PLTM.EQ.0.AND.PLTZP.EQ.0) GO TO 6
C
PLOT MASS OR ROLL INERTIA
C
IF (TORVB) 1,2
1 CALL NOTATE (0.,1.5,0.14,17HROLL INERTIA PLOT,90.,17)
GO TO 3
2 CALL NOTATE (0.,1.5,0.14,9HMASS PLOT,90.,9)

```

```

000133   3   CALL CALPLT (5.,0.,-3)                                M  42
000136                                         CALL GRID (0.,0.,1.,1.,INCHX,4)      M  43
000142                                         CALL AXES (0.,0.,0.,XINCH,XMIN,DX,1.,2.,18HSCALE OF X, INCHES,.125
1,-18)                                              M  44
                                                M  45
000155   4   IF (NMZP.EQ.1) 4,5                                M  46
000162   4   CALL ASCALE (MAS,4.,NSUM,1,20.)                  M  47
000166                                         MZRMIN=MAS(NSUM+1)          M  48
000170                                         DMZR=MAS(NSUM+2)          M  49
000172   5   CALL AXES (0.,0.,90.,4.,MZRMIN,DMZR,1.,2.,1H ,.125,1)    M  50
000205                                         CALL LINEX (X,MAS,NOMBR,NS,XMIN,DX,MZRMIN,DMZR,XP,YP)    M  51
000217                                         CALL CALPLT (12.,0.,-3)        M  52
000222                                         NPLOT=NPLOT+1            M  53
000224   6   IF (PLTAE.EQ.0.AND.PLTGJ.EQ.0) GO TO 9       M  54
C                                         M  55
C                                         PLOT STIFFNESS COEFFICIENT      M  56
C                                         M  57
000232                                         CALL NOTATE (0.,1.5,0.14,20HSTIFFNESS COEF. PLOT,90.,20)  M  58
000236                                         CALL CALPIT (5.,0.,-3)        M  59
000241                                         CALL GRID (0.,0.,1.,1.,INCHX,4)      M  60
000245                                         CALL AXES (0.,0.,0.,XINCH,XMIN,DX,1.,2.,18HSCALE OF X, INCHES,.125
1,-18)                                              M  61
                                                M  62
000260   7   IF (NMAEJG.EQ.1) 7,8                                M  63
000265   7   CALL ASCALE (Y,4.,NSUM,1,20.)                  M  64
000271                                         AJMIN=Y(NSUM+1)          M  65
000273                                         DAEJG=Y(NSUM+2)          M  66
000275   8   CALL AXES (0.,0.,90.,4.,AJMIN,DAEJG,1.,2.,1H ,.125,1)    M  67
000310                                         CALL LINEX (X,Y,NOMBP,NS,XMIN,DX,AJMIN,DAEJG,XP,YP)    M  68
000322                                         CALL CALPLT (12.,0.,-3)        M  69
000325                                         NPLOT=NPLOT+1            M  70
000327   9   IF (PLTZ.EQ.0) GO TO 10                      M  71
C                                         M  72
C                                         ZETA PLOTS                 M  73
C                                         M  74
000330                                         CALL NOTATE (0.,1.5,.14,10HZETA PLOTS,90.,10)  M  75
000334                                         CALL CALPLT (5.,0.,-3)        M  76
000337                                         CALL AXES (0.,0.,0.,XINCH,XMIN,DX,1.,2.,18HSCALE OF X, INCHES,.125
1,-18)                                              M  77
                                                M  78
000352  10  DO 12 II=1,3                                M  79
000354                                         DO 11 J=1,NSUM          M  80
000355  11  CALL RECIN (3,1,2,Y(J),Y1(J,II),DY(J,II))      M  81
000370                                         IF (PLTZ.EQ.0) GO TO 12      M  82
000371                                         CALL GRID (0.,0.,1.,.5,INCHX,2)      M  83
000375                                         CALL AXES (0.,0.,90.,1.,-1.,2.,1.,2.,1H ,.125,1)    M  84

```

```

000410      CALL LINEX (X,Y,NOMBR,NS,XMTN,DX,-1.,2.,XP,YP)
000422      CALL CALPLT (C.,1.5,-3)
000425      NPLOT=NPLOT+1
000427 12      CONTINUE
000431      IF (PLTZ.NE.0) CALL CALPLT (12.,-4.5,-3)
000435      IF (PLTZPR.EQ.0) GO TO 17

C
C      ZETA PRIME PLOTS
C

000436      CALL NOTATE (0.,1.,.14,16HZETA PRIME PLOTS,90.,16)
000442      CALL CALPLT (C.,0.,-3)
000445      CALL AXES (0.,0.,0.,XINCH,XMIN,DX,1.,2.,18HSCALE OF X, INCHES,.125
1.,-18)
000460      DO 16 IT=1,3
000462      CALL GRID (0.,0.,1.,.5,INCHX,2)
000466      IF (NZAPP.EQ.1) 13,15
000473 13      DO 14 I=1,NSUM
000475 14      YP(I)=Y(I,I)
000504      CALL ASCALE (YP,1.,NSUM,1,20.)
000510      ZPMIN(1)=YP(NSUM+1)
000513      DZP(1)=YP(NSUM+2)
000515 15      CALL AXES (0.,0.,90.,1.,ZPMIN(1),DZP(1),1.,2.,1H ,.125,1)
000530      CALL LINEX (X,Y1(1,IT),NOMBR,NS,XMIN,DX,ZPMIN(1),DZP(1),XP,YP)
000547      CALL CALPLT (0.,1.5,-3)
000552      NPLOT=NPLOT+1
000554 16      CONTINUE
000556      CALL CALPLT (12.,-4.5,-3)
000560 17      IF (PLTT.EQ.0) RETURN

C
C      TENSION OR TORQUE PLOT
C

000563      IF (TORVB) 18,19
000565 18      CALL NOTATE (0.,1.,.14,12HTORQUE PLOTS,90.,12)
000571      GO TO 20
000572 19      CALL NOTATE (0.,1.,.14,13HTENSION PLOTS,90.,13)
000576 20      CALL CALPLT (C.,0.,-3)
000601      IF (NPLTT.EQ.1) 21,23
000606 21      K=0
000607      DO 22 IT=1,3
000611      DO 22 I=1,NSUM
000612      K=K+1
000614      YP(K)=DY(I,IT)
000620 22      CONTINUE

```

000624	CALL ASCALE (YP,4.,K,1,20.)	M 128
000627	TMIN=YP(K+1)	M 129
000631	DTT=YP(K+2)	M 130
000633 23	DO 27 TI=1,3	M 131
000635	IF (II.EQ.1.OR.MPLT.NE.0) 24,25	M 132
000643 24	CALL GRID (0.,0.,1.,1.,INCHX,4)	M 133
000647	CALL AXES (0.,0.,0.,XINCH,XMIN,DX,1.,2.,18HSCALE OF X, INCHES,.125 1,-78)	M 134
000652	CALL AXES (0.,0.,90.,4.,TMIN,DTT,1.,2.,1H ,.125,1)	M 135
000675 25	CALL LINEX (X,DY(1,II),NOMBR,NS,XMIN,DX,TMIN,DTT,XP,YP)	M 136
000711	IF (MPLT.NE.0) 26,27	M 137
000715 26	CALL CALPLT (12.,0.,-3)	M 138
000720	NPLOT=NPLOT+1	M 139
000722 27	CONTINUE	M 140
000724	IF (MPLT.NE.0) RETURN	M 141
000727	CALL CALPLT (12.,0.,-3)	M 142
000732	NPLOT=NPLOT+1	M 143
000724	RETURN	M 144
C		M 145
000736 28	FORMAT (8A10)	M 146
000736	END	M 147.
		M 148-

```

SUBROUTINE LINEX (X,Y,NOMBR,NS,XMIN,DX,YMIN,DY,XP,YP)
DIMENSION X(1), Y(1), NS(1), XP(1), YP(1)
      K=1
      M=0
      DO 2 I=1,NOMBR
      N=NS(I)
      L=0
      M=M+N
      DO 1 J=K,M
      L=L+1
      XP(L)=X(J)
      YP(L)=Y(J)
      CONTINUE
      XP(N+1)=XMIN
      XP(N+2)=DX
      YP(N+1)=YMIN
      YP(N+2)=DY
      CALL LINPLT (XP,YP,N,1,0,0,0,0)
      K=M+1
      2 CONTINUE
      RETURN
      END

```

000015		N	1
000015		N	2
000015		N	3
000017		N	4
000020		N	5
000022		N	6
000022		N	7
000024		N	8
000026		N	9
000030		N	10
000033		N	11
000036	1	N	12
000040		N	13
000042		N	14
000044		N	15
000046		N	16
000051		N	17
000061		N	18
000063	2	N	19
000071		N	20
000072		N	21
		N	22--

APPENDIX C

COMPUTER PRINTOUT OF TORSIONAL MODAL DATA

FOR NUMERICAL EXAMPLE

TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

PARAMETER CONTROLS - TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

RFJG = 1.00000E+07 NODODE = 4
RFZR = 1.00000E-03 NORMBR = 0
RFX = 1.00000E+02 NORN = 0
OMEGA SUBR = 1.00000E+03 NORDIS = T
DELX = 1.00000E+00

OMEGA = 5.00000E+00
ZRMOD = 1.00000E+00 DELONG = 2.50000E+01
JGMOD = 1.00000E+00 OMGTOL = 1.00000E-07

120

BOUNDARY CONDITIONS

JOINT	NBR1	NBRJ	NBRK	NBRL	I0JTRN	IJTRN	IKTRN	ILTRN
1	0	1	0	0	3	0	0	0
2	1	2	0	3	3	1	0	1
3	3	0	0	0	3	0	0	0
4	2	0	0	0	3	0	0	0

SPRING AND FLEXIBILITY CONSTANTS

JOINT	MBRI	MBRJ	MFRK	MBRL	TRNS	TRNFIJ	TRNFIK	TRN FIL
1	0	1	0	0	3.000E+00	0.	0.	0.
2	1	2	0	3	3.000E+00	0.	0.	0.
3	3	0	0	0	3.000E+00	0.	0.	0.
4	2	0	0	0	3.000E+00	0.	0.	0.

TABLE I
 PHYSICAL CHARACTERISTICS
 FOR TORSIONAL VIBRATIONS
 TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZR	JG
1- 3	0.000	2.20000E-04	6.20000E+06
1- 2	40.000	2.20000E-04	6.20000E+06
2- 1	40.000	2.20000E-04	6.20000E+06
2- 2	120.000	2.20000E-04	6.20000E+06
3- 1	40.000	4.50000E-06	7.66990E+04
3- 2	120.000	4.50000E-06	7.66990E+04

CENTER OF GRAVITY X = 60.26906
 TOTAL MASS = .02676
 S = 1.61280E+00

TABLE II

TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

T = 0.00 SEC

MODE 1

FREQUENCY CYCLES PER SECOND 409.141

FREQUENCY RADIANS PER SECND 2570.7072

STATION	X	ZETA	ZETA-PRIME	TOQUE	TORQUE-PRIME
1- 1	0.000	-5.58777E-03	0.	0.	8.12394E-10
1- 2	1.000	-5.58712E-03	1.31031E-05	8.12394E+00	8.12298E-10
1- 3	2.000	-5.58515E-03	2.42032E-06	1.624A0F+01	8.12012E-10
1- 4	3.000	-5.58188E-03	3.92971E-06	2.43642E+01	8.11537E-10
1- 5	4.000	-5.57729E-03	5.23818E-06	3.24767E+01	8.10870F-10
1- 6	5.000	-5.57140E-03	6.54542E-06	4.05814E+01	8.10013E-10
1- 7	6.000	-5.56420E-03	7.85112E-06	4.86770E+01	8.08967E-10
1- 8	7.000	-5.55570E-03	9.15499E-06	5.67609E+01	8.07730E-10
1- 9	8.000	-5.54589E-03	1.04557E-05	6.48316E+01	8.06305E-10
1- 10	9.000	-5.53478E-03	1.17580E-05	7.28870F+01	8.04690E-10
1- 11	10.000	-5.52239E-03	1.30F25E-05	8.09254E+01	8.02886E-10
1- 12	11.000	-5.50868E-03	1.43459E-05	8.89448E+01	8.00895F-10
1- 13	12.000	-5.49369E-03	1.56360E-05	9.69432E+01	7.98715E-10
1- 14	13.000	-5.47741E-03	1.69224E-05	1.04919E+02	7.96348F-10
1- 15	14.000	-5.45984E-03	1.82049E-05	1.12870E+02	7.93794E-10
1- 16	15.000	-5.44100E-03	1.94831E-05	1.20795E+02	7.91055E-10
1- 17	16.000	-5.42298E-03	2.07567E-05	1.28691E+02	7.88129E-10
1- 18	17.000	-5.39948E-02	2.20254E-05	1.36558E+02	7.85019E-10
1- 19	18.000	-5.37683E-03	2.32890E-05	1.44392E+02	7.81725E-10
1- 20	19.000	-5.35291F-03	2.45471E-05	1.52192E+02	7.78247E-10
1- 21	20.000	-5.32773E-03	2.57995E-05	1.59957E+02	7.74587E-10
1- 22	21.000	-5.30131E-03	2.70458E-05	1.67684E+02	7.70745E-10
1- 23	22.000	-5.27364E-03	2.82857E-05	1.75372E+02	7.66723E-10
1- 24	23.000	-5.24474E-03	2.95191E-05	1.83018E+02	7.62521E-10

TABLE II, MODE 1
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
1- 25	24.000	-5.21460E-03	3.07455E-05	1.90522E+02	7.58139E-10
1- 26	25.000	-5.18325E-03	3.19447E-05	1.98181E+02	7.53580E-10
1- 27	26.000	-5.15067E-03	3.31764E-05	2.05694E+02	7.48845E-10
1- 28	27.000	-5.11689E-03	3.43803E-05	2.13158E+02	7.43934E-10
1- 29	28.000	-5.08191E-03	3.55762E-05	2.20572E+02	7.38848E-10
1- 30	29.000	-5.04574E-03	3.67637E-05	2.27935E+02	7.33500E-10
1- 31	30.000	-5.00850E-03	3.79426E-05	2.35244E+02	7.28150E-10
1- 32	31.000	-4.96990E-03	3.91120E-05	2.42498E+02	7.22553E-10
1- 33	32.000	-4.93016E-03	4.02734E-05	2.49695E+02	7.16789E-10
1- 34	33.000	-4.88931E-03	4.10240E-05	2.56834E+02	7.10940E-10
1- 35	34.000	-4.84721E-03	4.20067E-05	2.63912E+02	7.04760E-10
1- 36	35.000	-4.80410E-03	4.36932E-05	2.70920E+02	6.98468E-10
1- 37	36.000	-4.76991E-03	4.48194E-05	2.77801E+02	6.92033E-10
1- 38	37.000	-4.71494E-03	4.59209E-05	2.84769E+02	6.85493E-10
1- 39	38.000	-4.66880E-03	4.71707E-05	2.91590E+02	6.78970E-10
1- 40	39.000	-4.62044E-03	4.81190E-05	2.98342E+02	6.71761E-10
1- 41	40.000	-4.57181E-03	4.91974E-05	3.05029E+02	6.64586E-10
2- 1	40.000	-4.57101E-03	-1.94391E-04	3.120522E+03	6.00000E-10
2- 2	41.000	-4.57637E-03	-1.93296E-04	-1.19844E+02	6.92070E-10
2- 3	42.000	-4.59484E-03	-1.92136E-04	-1.19127E+03	7.20892E-10
2- 4	43.000	-4.61499E-03	-1.90971E-04	-1.18402E+03	7.48740E-10
2- 5	44.000	-4.64039E-03	-1.89741E-04	-1.17629E+03	7.76421E-10
2- 6	45.000	-4.62946E-03	-1.88466E-04	-1.16849E+03	8.03916E-10
2- 7	46.000	-4.71728E-03	-1.87147E-04	-1.16731E+03	8.31222E-10
2- 8	47.000	-4.70276E-03	-1.85785E-04	-1.15186E+03	8.58334E-10
2- 9	48.000	-4.70884E-03	-1.84378E-04	-1.14315E+03	8.85244E-10
2- 10	49.000	-4.27251E-03	-1.82929E-04	-1.13416E+03	9.11947E-10
2- 11	50.000	-4.45471E-03	-1.81437E-04	-1.12491E+03	9.38435E-10
2- 12	51.000	-4.63539E-03	-1.79902E-04	-1.11539E+03	9.64704E-10
2- 13	52.000	-4.81451E-03	-1.78325E-04	-1.10561E+03	9.90746E-10
2- 14	53.000	-4.99204E-03	-1.76706E-04	-1.09558E+03	1.01656E-09
2- 15	54.000	-5.16792E-03	-1.75046E-04	-1.08528E+03	1.04213E-09
2- 16	55.000	-5.34213E-03	-1.73344E-04	-1.07473E+03	1.06746E-09
2- 17	56.000	-5.51461E-03	-1.71602E-04	-1.06393E+03	1.09253E-09

TABLE II, MODE 1
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TOQUE	TORQUE-PRIME
2- 18	57.000	-7.68533E-03	-1.69820E-04	-1.05280E+03	1.11735E-09
2- 19	58.000	-7.85425E-03	-1.67998E-04	-1.04159E+03	1.14191E-09
2- 20	59.000	-8.02132E-03	-1.65134E-04	-1.03004E+03	1.16620E-09
2- 21	60.000	-8.18652E-03	-1.64234E-04	-1.01826E+03	1.19022E-09
2- 22	61.000	-8.34980E-03	-1.62297E-04	-1.00624E+03	1.21396E-09
2- 23	62.000	-8.51112E-03	-1.60320E-04	-9.93982E+02	1.23741E-09
2- 24	63.000	-8.67044E-03	-1.58205E-04	-9.81492E+02	1.26058E-09
2- 25	64.000	-8.82773E-03	-1.55253E-04	-9.68771E+02	1.28344E-09
2- 26	65.000	-8.98255E-03	-1.54165E-04	-9.55823E+02	1.30601E-09
2- 27	66.000	-9.13606E-03	-1.52041E-04	-9.42651E+02	1.32827E-09
2- 28	67.000	-9.28703E-03	-1.49880E-04	-9.29258E+02	1.35022E-09
2- 29	68.000	-9.43582E-03	-1.47685E-04	-9.15647E+02	1.37189E-09
2- 30	69.000	-9.58240E-03	-1.45455E-04	-9.01821E+02	1.39316E-09
2- 31	70.000	-9.72673E-03	-1.43191E-04	-8.87783E+02	1.41415E-09
2- 32	71.000	-9.86878E-03	-1.40893E-04	-8.73538E+02	1.43480E-09
2- 33	72.000	-1.00085E-02	-1.38562E-04	-8.59087E+02	1.45512E-09
2- 34	73.000	-1.01459E-02	-1.36199E-04	-8.44436E+02	1.47509E-09
2- 35	74.000	-1.02809E-02	-1.33804E-04	-8.29586E+02	1.49472E-09
2- 36	75.000	-1.04135E-02	-1.31378E-04	-8.14541E+02	1.51400E-09
2- 37	76.000	-1.05437E-02	-1.28920E-04	-7.99306E+02	1.53292E-09
2- 38	77.000	-1.06714E-02	-1.25433E-04	-7.83883E+02	1.55148E-09
2- 39	78.000	-1.07965E-02	-1.23915E-04	-7.68276E+02	1.56968E-09
2- 40	79.000	-1.09192E-02	-1.21369E-04	-7.52489E+02	1.58752E-09
2- 41	80.000	-1.10393E-02	-1.18794E-04	-7.36526E+02	1.60498E-09
2- 42	81.000	-1.1168E-02	-1.16192E-04	-7.20390E+02	1.62206E-09
2- 43	82.000	-1.12717E-02	-1.13562E-04	-7.04085E+02	1.63876E-09
2- 44	83.000	-1.13839E-02	-1.10906E-04	-6.87514E+02	1.65508E-09
2- 45	84.000	-1.14935E-02	-1.08223E-04	-6.70983E+02	1.67101E-09
2- 46	85.000	-1.16002E-02	-1.05515E-04	-6.54194E+02	1.68655E-09
2- 47	86.000	-1.17045E-02	-1.02783E-04	-6.37252E+02	1.70169E-09
2- 48	87.000	-1.18059E-02	-1.00026E-04	-6.20160E+02	1.71643E-09
2- 49	88.000	-1.19046E-02	-9.72457E-05	-6.02923E+02	1.73078E-09
2- 50	89.000	-1.20004E-02	-9.44427E-05	-5.85545E+02	1.74471E-09
2- 51	90.000	-1.20934E-02	-9.16176E-05	-5.68029E+02	1.75824E-09

TABLE II, MODE 1
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
2- 52	91.000	-1.21836E-02	-8.87710E-05	-5.50380E+02	1.77135E-09
2- 53	92.000	-1.22710E-02	-8.59036E-05	-5.32402E+02	1.78405E-09
2- 54	93.000	-1.23554E-02	-8.30160E-05	-5.14599E+02	1.79633E-09
2- 55	94.000	-1.24370E-02	-8.01090E-05	-4.96676E+02	1.80819E-09
2- 56	95.000	-1.25157E-02	-7.71831E-05	-4.78535E+02	1.81962E-09
2- 57	96.000	-1.25914E-02	-7.42392E-05	-4.60283E+02	1.83063E-09
2- 58	97.000	-1.26641E-02	-7.12779E-05	-4.41923E+02	1.84121E-09
2- 59	98.000	-1.27339E-02	-6.82998E-05	-4.23459E+02	1.85136E-09
2- 60	99.000	-1.28007E-02	-6.53057E-05	-4.04896E+02	1.86107E-09
2- 61	100.000	-1.28645E-02	-6.22964E-05	-3.86237E+02	1.87035E-09
2- 62	101.000	-1.29253E-02	-5.93724E-05	-3.67489E+02	1.87919E-09
2- 63	102.000	-1.29831E-02	-5.62365E-05	-3.48654E+02	1.88758E-09
2- 64	103.000	-1.30378E-02	-5.31834E-05	-3.29737E+02	1.89554E-09
2- 65	104.000	-1.30845E-02	-5.01198E-05	-3.10742E+02	1.90305E-09
2- 66	105.000	-1.31380E-02	-4.70445E-05	-2.91676E+02	1.91011E-09
2- 67	106.000	-1.31835E-02	-4.39582E-05	-2.72541E+02	1.91673E-09
2- 68	107.000	-1.32260E-02	-4.08615E-05	-2.5342E+02	1.92289E-09
2- 69	108.000	-1.32653E-02	-3.77553E-05	-2.34083E+02	1.92861E-09
2- 70	109.000	-1.33015E-02	-3.46402E-05	-2.14769E+02	1.93387E-09
2- 71	110.000	-1.33345E-02	-3.15170E-05	-1.95606E+02	1.93868E-09
2- 72	111.000	-1.33645E-02	-2.83844E-05	-1.75959E+02	1.94304E-09
2- 73	112.000	-1.33913E-02	-2.52492E-05	-1.56545E+02	1.94693E-09
2- 74	113.000	-1.34150E-02	-2.21060E-05	-1.37057E+02	1.95038E-09
2- 75	114.000	-1.34355E-02	-1.89576E-05	-1.17537E+02	1.95336E-09
2- 76	115.000	-1.34529E-02	-1.58048E-05	-9.79899E+01	1.95589E-09
2- 77	116.000	-1.34671E-02	-1.26483E-05	-7.84195E+01	1.95796E-09
2- 78	117.000	-1.34782E-02	-9.49883E-06	-5.88307E+01	1.95957E-09
2- 79	118.000	-1.34861E-02	-6.32712E-06	-3.92281E+01	1.96072E-09
2- 80	119.000	-1.34909E-02	-3.16293E-06	-1.96164E+01	1.96141E-09
2- 81	120.000	-1.34925E-02	-2.00118E-20	-1.24073E-13	1.96164E-09
3- 1	40.000	-4.57181E-03	1.96906E-02	1.51025E+03	1.35958E-11
3- 2	41.000	1.51197E-02	1.96885E-02	1.51009E+03	-4.49635E-11
3- 3	42.000	3.48053E-02	1.96789E-02	1.50935E+03	-1.03505E-10
3- 4	43.000	5.44774E-02	1.96616E-02	1.50802E+03	-1.62007E-10

TABLE II, MODE 1
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
2- 5	44.000	7.41284E-02	1.96346E-02	1.50411E+03	-2.20446E-10
2- 6	45.000	9.37507E-02	1.96041E-02	1.50361E+03	-2.78800E-10
3- 7	46.000	1.13337E-01	1.95639E-02	1.50053E+03	-3.37045E-10
2- 8	47.000	1.32879E-01	1.95162E-02	1.49687E+03	-3.95160E-10
2- 9	48.000	1.52369E-01	1.94609E-02	1.49263E+03	-4.53121E-10
3- 10	49.000	1.71800E-01	1.93280E-02	1.48781E+03	-5.10907E-10
3- 11	50.000	1.91165E-01	1.93277E-02	1.48241E+03	-5.68494E-10
2- 12	51.000	2.10456E-01	1.92498E-02	1.47644E+03	-6.25862E-10
2- 13	52.000	2.29665E-01	1.91645E-02	1.46990E+03	-6.82986E-10
2- 14	53.000	2.48788E-01	1.90717E-02	1.46278E+03	-7.39846E-10
3- 15	54.000	2.67808E-01	1.89714E-02	1.45510E+03	-7.95419E-10
3- 16	55.000	2.86728E-01	1.88640E-02	1.44685E+03	-8.52682E-10
3- 17	56.000	3.05536E-01	1.87492E-02	1.43804E+03	-9.08616E-10
3- 18	57.000	3.24226E-01	1.86271E-02	1.42868E+03	-9.64197E-10
3- 19	58.000	3.42790E-01	1.84978E-02	1.41877E+03	-1.01940E-09
2- 20	59.000	3.61222E-01	1.83613E-02	1.40829E+03	-1.07422E-09
2- 21	60.000	3.79517E-01	1.82177E-02	1.39728E+03	-1.12861E-09
3- 22	61.000	3.97657E-01	1.80670E-02	1.38572E+03	-1.18257E-09
3- 23	62.000	4.15647E-01	1.79093E-02	1.37363E+03	-1.23607E-09
3- 24	63.000	4.33476E-01	1.77447E-02	1.36100E+03	-1.28909E-09
2- 25	64.000	4.51135E-01	1.75732E-02	1.34784E+03	-1.34141E-09
3- 26	65.000	4.68422E-01	1.73948E-02	1.33417E+03	-1.39361E-09
3- 27	66.000	4.85926E-01	1.72098E-02	1.31997E+03	-1.44507E-09
3- 28	67.000	5.03041E-01	1.70180E-02	1.30527E+03	-1.49597E-09
2- 29	68.000	5.19962E-01	1.68197E-02	1.29005E+03	-1.54628E-09
3- 30	69.000	5.36681E-01	1.66148E-02	1.27434E+03	-1.59600E-09
2- 31	70.000	5.53192E-01	1.64035E-02	1.25813E+03	-1.64510E-09
3- 32	71.000	5.69488E-01	1.61858E-02	1.24144E+03	-1.69257E-09
2- 33	72.000	5.85563E-01	1.59619E-02	1.22426E+03	-1.74137E-09
2- 34	73.000	6.01412E-01	1.57318E-02	1.20641E+03	-1.78850E-09
3- 35	74.000	6.17027E-01	1.54955E-02	1.18849E+03	-1.83494E-09
3- 36	75.000	6.32403E-01	1.52533E-02	1.16991E+03	-1.88067E-09
3- 37	76.000	6.47533E-01	1.50205E-02	1.15088E+03	-1.92566E-09
3- 38	77.000	6.62413E-01	1.47511E-02	1.13140E+03	-1.96991E-09

TABLE II, MODE 1
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
3- 39	78.000	6.77026E-01	1.44914E-02	1.11148E+03	-2.01340E-09
3- 40	79.000	6.91256E-01	1.42261E-02	1.09112E+03	-2.05610E-09
3- 41	80.000	7.05488E-01	1.39553E-02	1.07036E+03	-2.09801E-09
3- 42	81.000	7.19307E-01	1.36791E-02	1.04917E+03	-2.13910E-09
3- 43	82.000	7.32846E-01	1.33975E-02	1.02758E+03	-2.17937E-09
3- 44	83.000	7.46102E-01	1.31108E-02	1.00558E+03	-2.21879E-09
3- 45	84.000	7.59068E-01	1.28189E-02	9.83200E+02	-2.25735E-09
3- 46	85.000	7.71740E-01	1.25221E-02	9.60436E+02	-2.29503E-09
3- 47	86.000	7.84112E-01	1.22205E-02	9.37299E+02	-2.33182E-09
3- 48	87.000	7.96181E-01	1.19141E-02	9.13799E+02	-2.36771E-09
3- 49	88.000	8.07940E-01	1.16031E-02	8.89945E+02	-2.40269E-09
3- 50	89.000	8.19387E-01	1.12876E-02	8.65746E+02	-2.43673E-09
3- 51	90.000	8.30514E-01	1.09677E-02	8.41211E+02	-2.46982E-09
3- 52	91.000	8.41222E-01	1.06435E-02	8.16349E+02	-2.50196E-09
3- 53	92.000	8.51803E-01	1.03153E-02	7.91171E+02	-2.53313E-09
3- 54	93.000	8.61957E-01	9.99301E-03	7.65687E+02	-2.56331E-09
3- 55	94.000	8.71769E-01	9.64687E-03	7.39905E+02	-2.59250E-09
3- 56	95.000	8.81246E-01	9.30699E-03	7.13837E+02	-2.62069E-09
3- 57	96.000	8.90383E-01	8.96350E-03	6.87492E+02	-2.64786E-09
3- 58	97.000	8.99174E-01	8.61644E-03	6.60880E+02	-2.67400E-09
3- 59	98.000	9.07616E-01	8.25623E-03	6.34012E+02	-2.69910E-09
3- 60	99.000	9.15705E-01	7.91272E-03	6.04898E+02	-2.72316E-09
3- 61	100.000	9.23441E-01	7.55614E-03	5.79548E+02	-2.74617E-09
3- 62	101.000	9.30818E-01	7.19663E-03	5.51974E+02	-2.76811E-09
3- 63	102.000	9.37834E-01	6.83433E-03	5.24186E+02	-2.78897E-09
3- 64	103.000	9.44487E-01	6.46938E-03	4.96195E+02	-2.80875E-09
3- 65	104.000	9.50773E-01	6.10192E-03	4.68011E+02	-2.82745E-09
3- 66	105.000	9.56691E-01	5.73209E-03	4.39546E+02	-2.84505E-09
3- 67	106.000	9.62238E-01	5.36005E-03	4.11110E+02	-2.86154E-09
3- 68	107.000	9.67411E-01	4.98592E-03	3.82415E+02	-2.87693E-09
3- 69	108.000	9.72209E-01	4.60984E-03	3.53572E+02	-2.89120E-09
3- 70	109.000	9.76631E-01	4.23201E-03	3.24591E+02	-2.90434E-09
3- 71	110.000	9.80873E-01	3.85253E-03	2.95485E+02	-2.91637E-09
3- 72	111.000	9.84336E-01	3.47154E-03	2.66264E+02	-2.92726E-09

TABLE II, MODE 1
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
3- 73	112.000	9.87617E-01	3.08922E-03	2.36940E+02	-2.93701E-09
3- 74	113.000	9.90514E-01	2.70569E-03	2.07524E+02	-2.94563E-09
3- 75	114.000	9.93028E-01	2.32111E-03	1.78027E+02	-2.95311E-09
3- 76	115.000	9.95157E-01	1.93564E-03	1.48461E+02	-2.95944E-09
3- 77	116.000	9.96899E-01	1.54941E-03	1.18838E+02	-2.96462E-09
3- 78	117.000	9.98255E-01	1.16258E-03	8.91691E+01	-2.96865E-09
3- 79	118.000	9.99225E-01	7.75307E-04	5.94653E+01	-2.97153E-09
3- 80	119.000	9.99806E-01	3.87729E-04	2.97364E+01	-2.97326E-09
3- 81	120.000	1.00000E+00	-1.29413E-17	-9.92588E-13	-2.97384E-09

129 TOTAL NO. STATIONS = 203

TABLE II
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

$T = 0.00 \text{ SEC}$

MODE 2

FREQUENCY CYCLES PER SECOND 699.821

FREQUENCY RADIANS PER SECOND 4397.1067

130

	STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
1-	1	0.000	1.00000E+00	0.	0.	-4.25360E-07
1-	2	1.000	9.99657E-01	-6.86065E-04	-4.25360E+03	-4.25214E-07
1-	3	2.000	9.98428E-01	-1.37144E-03	-8.0428E+03	-4.24776E-07
1-	4	3.000	9.96914E-01	-2.05631E-03	-1.27491E+04	-4.24047E-07
1-	5	4.000	9.94514E-01	-2.73955E-03	-1.69852E+04	-4.23027E-07
1-	6	5.000	9.91435E-01	-3.42092E-03	-2.12097E+04	-4.21717E-07
1-	7	6.000	9.87674E-01	-4.09993E-03	-2.54196E+04	-4.20117E-07
1-	8	7.000	9.83235E-01	-4.77612E-03	-2.96120E+04	-4.18229E-07
1-	9	8.000	9.78122E-01	-5.44906E-03	-3.37841E+04	-4.16054E-07
1-	10	9.000	9.72337E-01	-6.11824E-03	-3.79331E+04	-4.13593E-07
1-	11	10.000	9.65885E-01	-6.78323E-03	-4.20560E+04	-4.10849E-07
1-	12	11.000	9.58771E-01	-7.44356E-03	-4.61501E+04	-4.07823E-07
1-	13	12.000	9.50998E-01	-8.09879E-03	-5.02125E+04	-4.04517E-07
1-	14	13.000	9.42573E-01	-8.74845E-03	-5.42404E+04	-4.00933E-07
1-	15	14.000	9.33501E-01	-9.39212E-03	-5.82311E+04	-3.97074E-07
1-	16	15.000	9.23789E-01	-1.00293E-02	-6.21819E+04	-3.92943E-07
1-	17	16.000	9.13443E-01	-1.06597E-02	-6.60900E+04	-3.88542E-07
1-	18	17.000	9.02470E-01	-1.12827E-02	-6.99528E+04	-3.83875E-07
1-	19	18.000	8.90878E-01	-1.18980E-02	-7.37675E+04	-3.78944E-07
1-	20	19.000	8.78674E-01	-1.25051E-02	-7.75316E+04	-3.73753E-07
1-	21	20.000	8.65867E-01	-1.31036E-02	-8.12426E+04	-3.68305E-07
1-	22	21.000	8.52467E-01	-1.36932E-02	-8.48978E+04	-3.62605E-07
1-	23	22.000	8.38481E-01	-1.42733E-02	-8.84947E+04	-3.56656E-07
1-	24	23.000	8.23920E-01	-1.48437E-02	-9.20309E+04	-3.50463E-07

TABLE II, MODE 2
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
1- 25	24.000	8.08794E-01	-1.54029E-02	-9.55040E+04	-3.44029E-07
1- 26	25.000	7.93113E-01	-1.59535E-02	-9.89115E+04	-3.37358E-07
1- 27	26.000	7.76887E-01	-1.64921E-02	-1.02251E+05	-3.30457E-07
1- 28	27.000	7.60128E-01	-1.70195E-02	-1.05521E+05	-3.23328E-07
1- 29	28.000	7.42848E-01	-1.75351E-02	-1.08718E+05	-3.15978E-07
1- 30	29.000	7.25058E-01	-1.80387E-02	-1.11840E+05	-3.08411E-07
1- 31	30.000	7.06771E-01	-1.85300E-02	-1.14886E+05	-3.00632E-07
1- 32	31.000	6.87998E-01	-1.90085E-02	-1.17853E+05	-2.92647E-07
1- 33	32.000	6.68754E-01	-1.94740E-02	-1.20739E+05	-2.84461E-07
1- 34	33.000	6.49050E-01	-1.99261E-02	-1.23542E+05	-2.76080E-07
1- 35	34.000	6.28902E-01	-2.03646E-02	-1.26261E+05	-2.67510E-07
1- 36	35.000	6.08321E-01	-2.07891E-02	-1.28892E+05	-2.58756E-07
1- 37	36.000	5.87324E-01	-2.11993E-02	-1.31438E+05	-2.49824E-07
1- 38	37.000	5.65923E-01	-2.15950E-02	-1.33889E+05	-2.40721E-07
1- 39	38.000	5.44134E-01	-2.19758E-02	-1.36250E+05	-2.31453E-07
1- 40	39.000	5.21971E-01	-2.23416E-02	-1.38518E+05	-2.22026E-07
1- 41	40.000	4.99451E-01	-2.26920E-02	-1.40691E+05	-2.12446E-07
2- 1	40.000	4.99451E-01	-2.25924E-02	-1.40072E+05	-2.12446E-07
2- 2	41.000	4.76687E-01	-2.29273E-02	-1.42149E+05	-2.02764E-07
2- 3	42.000	4.53596E-01	-2.32464E-02	-1.44128E+05	-1.92942E-07
2- 4	43.000	4.30154E-01	-2.35497E-02	-1.46008E+05	-1.82987E-07
2- 5	44.000	4.06497E-01	-2.38357E-02	-1.47788E+05	-1.72908E-07
2- 6	45.000	3.82521E-01	-2.41074E-02	-1.49466E+05	-1.62709E-07
2- 7	46.000	3.58282E-01	-2.43616E-02	-1.51042E+05	-1.52399E-07
2- 8	47.000	3.33798E-01	-2.45990E-02	-1.52514E+05	-1.41984E-07
2- 9	48.000	3.09084E-01	-2.48196E-02	-1.53882E+05	-1.31472E-07
2- 10	49.000	2.84158E-01	-2.50232E-02	-1.55144E+05	-1.20870E-07
2- 11	50.000	2.59038E-01	-2.52095E-02	-1.56299E+05	-1.10184E-07
2- 12	51.000	2.33739E-01	-2.53786E-02	-1.57347E+05	-9.94234E-08
2- 13	52.000	2.08281E-01	-2.55302E-02	-1.58287E+05	-8.85942E-08
2- 14	53.000	1.82679E-01	-2.56644E-02	-1.59119E+05	-7.77043E-08
2- 15	54.000	1.56952E-01	-2.57809E-02	-1.59842E+05	-6.67611E-08
2- 16	55.000	1.31117E-01	-2.58797E-02	-1.60454E+05	-5.57720E-08
2- 17	56.000	1.05192E-01	-2.59608E-02	-1.60957E+05	-4.47446E-08

TABLE II, MODE 2
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
2- 18	57.000	7.91955E-02	-2.60241E-02	-1.61349E+05	-3.36846E-08
2- 19	58.000	5.31443E-02	-2.60695E-02	-1.61631E+05	-2.26054E-08
2- 20	59.000	2.70565E-02	-2.60970E-02	-1.61801E+05	-1.15082E-08
2- 21	60.000	9.50263E-04	-2.61056E-02	-1.61841E+05	-4.04204E-10
2- 22	61.000	-2.51567E-02	-2.60983E-02	-1.61810E+05	1.07006E-08
2- 23	62.000	-5.12464E-02	-2.60721E-02	-1.61647E+05	2.17982E-08
2- 24	63.000	-7.71009E-02	-2.60280E-02	-1.41374E+05	3.28807E-08
2- 25	64.000	-1.03302E-01	-2.59660E-02	-1.60989E+05	4.39407E-08
2- 26	65.000	-1.29223E-01	-2.58863E-02	-1.60495E+05	5.49705E-08
2- 27	66.000	-1.56075E-01	-2.57887E-02	-1.59890E+05	6.59627E-08
2- 28	67.000	-1.80810E-01	-2.55725E-02	-1.59176E+05	7.69095E-08
2- 29	68.000	-2.06422E-01	-2.55406E-02	-1.58322E+05	8.78036E-08
2- 30	69.000	-2.31892E-01	-2.53902E-02	-1.57419E+05	9.86374E-08
2- 31	70.000	-2.57202E-01	-2.52224E-02	-1.56379E+05	1.09404E-07
2- 32	71.000	-2.82337E-01	-2.50373E-02	-1.55221E+05	1.20095E-07
2- 33	72.000	-3.07277E-01	-2.48350E-02	-1.52977E+05	1.30703E-07
2- 34	73.000	-3.32007E-01	-2.46157E-02	-1.52617E+05	1.41222E-07
2- 35	74.000	-3.56509E-01	-2.43794E-02	-1.51153E+05	1.51644E-07
2- 36	75.000	-3.80766E-01	-2.41265E-02	-1.49585E+05	1.61963E-07
2- 37	76.000	-4.04762E-01	-2.38570E-02	-1.47914E+05	1.72169E-07
2- 38	77.000	-4.28480E-01	-2.35712E-02	-1.46141E+05	1.82258E-07
2- 39	78.000	-4.51904E-01	-2.32691E-02	-1.44268E+05	1.92222E-07
2- 40	79.000	-4.75018E-01	-2.29511E-02	-1.42297E+05	2.02054E-07
2- 41	80.000	-4.97806E-01	-2.26173E-02	-1.40227E+05	2.11747E-07
2- 42	81.000	-5.20253E-01	-2.22680E-02	-1.38062E+05	2.21295E-07
2- 43	82.000	-5.42342E-01	-2.19034E-02	-1.35802E+05	2.30691E-07
2- 44	83.000	-5.64060E-01	-2.15239E-02	-1.33448E+05	2.39928E-07
2- 45	84.000	-5.85390E-01	-2.11295E-02	-1.31003E+05	2.49002E-07
2- 46	85.000	-6.06319E-01	-2.07206E-02	-1.28468E+05	2.57904E-07
2- 47	86.000	-6.26821E-01	-2.02976E-02	-1.25845E+05	2.66629E-07
2- 48	87.000	-6.46914E-01	-1.98606E-02	-1.23135E+05	2.75171E-07
2- 49	88.000	-6.66533E-01	-1.94099E-02	-1.20341E+05	2.83525E-07
2- 50	89.000	-6.85734E-01	-1.89460E-02	-1.17465E+05	2.91684E-07
2- 51	90.000	-7.04445E-01	-1.84690E-02	-1.14508E+05	2.99643E-07

TABLE II, MODE 2
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
2- 52	91.000	-7.22672E-01	-1.79794E-02	-1.11472E+05	3.07396E-07
2- 53	92.000	-7.40404E-01	-1.74774E-02	-1.08360E+05	3.14938E-07
2- 54	93.000	-7.57627E-01	-1.69634E-02	-1.05177E+05	3.22254E-07
2- 55	94.000	-7.74330E-01	-1.64378E-02	-1.01911E+05	3.29369E-07
2- 56	95.000	-7.90503E-01	-1.59010E-02	-9.85860E+04	3.36248E-07
2- 57	96.000	-8.06133E-01	-1.53532E-02	-9.51897E+04	3.42897E-07
2- 58	97.000	-8.21209E-01	-1.47949E-02	-9.17281E+04	3.49310E-07
2- 59	98.000	-8.35722E-01	-1.42254E-02	-8.82775E+04	3.55483E-07
2- 60	99.000	-8.49552E-01	-1.36481E-02	-8.46184E+04	3.61412E-07
2- 61	100.000	-8.63019E-01	-1.30605E-02	-8.09752E+04	3.67094E-07
2- 62	101.000	-8.75783E-01	-1.24640E-02	-7.72766E+04	3.72523E-07
2- 63	102.000	-8.87947E-01	-1.18588E-02	-7.35248E+04	3.77697E-07
2- 64	103.000	-8.99501E-01	-1.12456E-02	-6.97226E+04	3.82612E-07
2- 65	104.000	-9.10428E-01	-1.06244E-02	-6.58724E+04	3.87264E-07
2- 66	105.000	-9.20750E-01	-9.99635E-03	-6.19774E+04	3.91650E-07
2- 67	106.000	-9.30431E-01	-9.26123E-03	-5.80394E+04	3.95768E-07
2- 68	107.000	-9.29472E-01	-8.71968E-03	-5.40620E+04	3.99614E-07
2- 69	108.000	-9.47870E-01	-8.07215E-03	-5.00473E+04	4.03186E-07
2- 70	109.000	-9.55617E-01	-7.41908E-03	-4.59983E+04	4.06481E-07
2- 71	110.000	-9.62709E-01	-6.75092E-03	-4.19177E+04	4.09498E-07
2- 72	111.000	-9.49139E-01	-6.09812E-03	-3.78083E+04	4.12233E-07
2- 73	112.000	-9.74905E-01	-5.41114E-03	-3.36730E+04	4.14686E-07
2- 74	113.000	-9.80002E-01	-4.76042E-03	-2.95146E+04	4.16854E-07
2- 75	114.000	-9.84426E-01	-4.08645E-03	-2.52360E+04	4.18735E-07
2- 76	115.000	-9.88175E-01	-3.40967E-03	-2.11399E+04	4.20330E-07
2- 77	116.000	-9.91245E-01	-2.73054E-03	-1.69294E+04	4.21636E-07
2- 78	117.000	-9.93636E-01	-2.04955E-03	-1.27072E+04	4.22653E-07
2- 79	118.000	-9.95244E-01	-1.36715E-03	-8.47632E+03	4.23380E-07
2- 80	119.000	-9.96370E-01	-6.83809E-04	-4.23961E+03	4.23816E-07
2- 81	120.000	-9.96712E-01	3.04416E-16	1.88738E-09	4.23962E-07
3- 1	40.000	4.99451E-01	-8.05622E-03	-6.17911E+02	-4.34549E-09
3- 2	41.000	4.91111E-01	-8.61831E-03	-6.61016E+02	-4.27293E-09
3- 3	42.000	4.82214E-01	-9.17053E-03	-7.03370E+02	-4.19553E-09
3- 4	43.000	4.72770E-01	-9.71234E-03	-7.44927E+02	-4.11336E-09

TABLE II, MODE 2
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
3- 5	44.000	4.62790E-01	-1.02431E-02	-7.85638E+02	-4.02652E-09
3- 6	45.000	4.52284E-01	-1.07623E-02	-8.25457E+02	-3.93512E-09
3- 7	46.000	4.41265E-01	-1.12693E-02	-8.64340E+02	-3.83925E-09
3- 8	47.000	4.29746E-01	-1.17634E-02	-9.02243E+02	-3.73902E-09
3- 9	48.000	4.17739E-01	-1.22442E-02	-9.39121E+02	-3.63455E-09
3- 10	49.000	4.05257E-01	-1.27112E-02	-9.74934E+02	-3.52696E-09
3- 11	50.000	3.92216E-01	-1.31637E-02	-1.00964E+03	-3.41337E-09
3- 12	51.000	3.78930E-01	-1.36012E-02	-1.04320E+03	-3.29690E-09
3- 13	52.000	3.65114E-01	-1.40234E-02	-1.07558E+03	-3.17669E-09
3- 14	53.000	3.50883E-01	-1.44296E-02	-1.10574E+03	-3.05288E-09
3- 15	54.000	3.36255E-01	-1.48194E-02	-1.13664E+03	-2.92560E-09
3- 16	55.000	3.21245E-01	-1.51925E-02	-1.16528E+03	-2.79500E-09
3- 17	56.000	3.05870E-01	-1.55483E-02	-1.19254E+03	-2.66124E-09
3- 18	57.000	2.90148E-01	-1.58844E-02	-1.21847E+03	-2.52445E-09
3- 19	58.000	2.74097E-01	-1.62066E-02	-1.24303E+03	-2.38480E-09
3- 20	59.000	2.57735E-01	-1.65083E-02	-1.26617E+03	-2.24244E-09
3- 21	60.000	2.41081E-01	-1.67913E-02	-1.28788E+03	-2.09753E-09
3- 22	61.000	2.24153E-01	-1.70552E-02	-1.30812E+03	-1.95025E-09
3- 23	62.000	2.06970E-01	-1.72998E-02	-1.32488E+03	-1.80075E-09
3- 24	63.000	1.89553E-01	-1.75248E-02	-1.34414E+03	-1.64922E-09
3- 25	64.000	1.71921E-01	-1.77299E-02	-1.35987E+03	-1.49580E-09
3- 26	65.000	1.54093E-01	-1.79149E-02	-1.37405E+03	-1.34070E-09
3- 27	66.000	1.36091E-01	-1.80795E-02	-1.38668E+03	-1.18407E-09
3- 28	67.000	1.17934E-01	-1.82236E-02	-1.39773E+03	-1.02609E-09
3- 29	68.000	9.96439E-02	-1.83471E-02	-1.40720E+03	-8.66956E-10
3- 30	69.000	8.12403E-02	-1.84497E-02	-1.41507E+03	-7.06835E-10
3- 31	70.000	6.27446E-02	-1.85314E-02	-1.42134E+03	-5.45911E-10
3- 32	71.000	4.41775E-02	-1.85921E-02	-1.42599E+03	-3.84368E-10
3- 33	72.000	2.55604E-02	-1.85316E-02	-1.42907E+03	-2.22389E-10
3- 34	73.000	6.91424E-03	-1.85501E-02	-1.43044E+03	-6.01577E-11
3- 35	74.000	-1.17397E-02	-1.86473E-02	-1.43023E+03	1.02142E-10
3- 36	75.000	-3.03804E-02	-1.86234E-02	-1.42840E+03	2.64326E-10
3- 37	76.000	-4.89866E-02	-1.85784E-02	-1.42495E+03	4.26210E-10
3- 38	77.000	-6.75372E-02	-1.85123E-02	-1.41987E+03	5.87611E-10

TABLE II, MODE 2
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
2- 39	78.000	-8.60112E-02	-1.84252E-02	-1.41319E+03	7.48345E-10
2- 40	79.000	-1.04388E-01	-1.83172E-02	-1.40493E+03	9.08229E-10
3- 41	80.000	-1.22646E-01	-1.81884E-02	-1.39502E+03	1.06708E-09
3- 42	81.000	-1.40764E-01	-1.80389E-02	-1.38357E+03	1.22473E-09
3- 43	82.000	-1.58723E-01	-1.78690E-02	-1.37054E+03	1.38098E-09
3- 44	83.000	-1.76502E-01	-1.76788E-02	-1.35595E+03	1.53567E-09
3- 45	84.000	-1.94081E-01	-1.74686E-02	-1.33982E+03	1.68861E-09
3- 46	85.000	-2.11440E-01	-1.72385E-02	-1.32218E+03	1.83964E-09
3- 47	86.000	-2.28558E-01	-1.69889E-02	-1.30303E+03	1.98878E-09
3- 48	87.000	-2.45417E-01	-1.67200E-02	-1.28240E+03	2.13527E-09
3- 49	88.000	-2.61958E-01	-1.64321E-02	-1.26032E+03	2.27953E-09
3- 50	89.000	-2.78282E-01	-1.61256E-02	-1.23581E+03	2.42120E-09
3- 51	90.000	-2.94249E-01	-1.58007E-02	-1.21190E+03	2.56013E-09
3- 52	91.000	-3.09883E-01	-1.54580E-02	-1.18561E+03	2.69615E-09
3- 53	92.000	-3.25166E-01	-1.50977E-02	-1.15798E+03	2.82912E-09
3- 54	93.000	-3.40079E-01	-1.47203E-02	-1.12903E+03	2.95887E-09
3- 55	94.000	-3.54406E-01	-1.43262E-02	-1.09880E+03	3.08527E-09
3- 56	95.000	-3.68731E-01	-1.39158E-02	-1.06733E+03	3.20816E-09
3- 57	96.000	-3.82438E-01	-1.34894E-02	-1.03464E+03	3.32742E-09
3- 58	97.000	-3.95711E-01	-1.30481E-02	-1.00078E+03	3.44290E-09
3- 59	98.000	-4.08534E-01	-1.25918E-02	-9.65781E+02	3.55447E-09
3- 60	99.000	-4.20894E-01	-1.21213E-02	-9.29689E+02	3.66201E-09
3- 61	100.000	-4.32777E-01	-1.16369E-02	-8.92541E+02	3.76540E-09
3- 62	101.000	-4.44168E-01	-1.11394E-02	-8.54381E+02	3.84451E-09
3- 63	102.000	-4.55056E-01	-1.06292E-02	-8.15251E+02	3.95923E-09
3- 64	103.000	-4.65427E-01	-1.01070E-02	-7.75196E+02	4.04947E-09
3- 65	104.000	-4.75270E-01	-9.57329E-03	-7.34262E+02	4.13511E-09
3- 66	105.000	-4.84574E-01	-9.02873E-03	-6.92495E+02	4.21606E-09
3- 67	106.000	-4.93328E-01	-8.47392E-03	-6.49941E+02	4.29222E-09
3- 68	107.000	-5.01522E-01	-7.90950E-03	-6.06650E+02	4.36351E-09
3- 69	108.000	-5.09147E-01	-7.33610E-03	-5.62671E+02	4.42985E-09
3- 70	109.000	-5.16194E-01	-6.75437E-03	-5.18054E+02	4.49117E-09
3- 71	110.000	-5.22636E-01	-6.16498E-03	-4.72848E+02	4.54739E-09
3- 72	111.000	-5.28524E-01	-5.56860E-03	-4.27106E+02	4.59845E-09

TABLE II, MODE 2
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
3- 73	112.000	-5.33793E-01	-4.96589E-03	-3.80879E+02	4.64429E-09
3- 74	113.000	-5.38456E-01	-4.35756E-03	-3.34220E+02	4.68486E-09
3- 75	114.000	-5.42508E-01	-3.74427E-03	-2.87182E+02	4.72012E-09
3- 76	115.000	-5.45945E-01	-3.12674E-03	-2.39818E+02	4.75002E-09
3- 77	116.000	-5.48762E-01	-2.50566E-03	-1.92182E+02	4.77453E-09
3- 78	117.000	-5.50956E-01	-1.88174E-03	-1.44327E+02	4.79362E-09
3- 79	118.000	-5.52526E-01	-1.25568E-03	-9.63094E+01	4.80727E-09
3- 80	119.000	-5.53468E-01	-6.28197E-04	-4.81821E+01	4.81547E-09
3- 81	120.000	-5.53782E-01	-2.71407E-17	-2.08167E-12	4.81821E-09

TOTAL NO. STATIONS = 203

TABLE II
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE
 $T = 0.00 \text{ SEC}$
MODE 3

FREQUENCY CYCLES PER SECOND 1221.945

FREQUENCY RADTANS PER SECOND 7677.7046

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
1- 1	0.000	1.93881E-02	0.	0.	-2.51432E-08
1- 2	1.000	1.93678E-02	-4.05535E-05	-2.51432E+02	-2.51169E-08
1- 3	2.000	1.93070E-02	-8.10221E-05	-5.02337E+02	-2.50380E-08
1- 4	3.000	1.92058E-02	-1.21321E-04	-7.52192E+02	-2.49067E-08
1- 5	4.000	1.90644E-02	-1.61347E-04	-1.00047E+03	-2.47234E-08
1- 6	5.000	1.88831E-02	-2.01074E-04	-1.24566E+03	-2.44882E-08
1- 7	6.000	1.86622E-02	-2.40361E-04	-1.49024E+03	-2.42019E-08
1- 8	7.000	1.84023E-02	-2.79145E-04	-1.73070E+03	-2.38649E-08
1- 9	8.000	1.81040E-02	-3.17345E-04	-1.98754E+03	-2.34779E-08
1- 10	9.000	1.77677E-02	-3.54880E-04	-2.20026E+03	-2.30418E-08
1- 11	10.000	1.73942E-02	-3.91673E-04	-2.42838E+03	-2.25575E-08
1- 12	11.000	1.69844E-02	-4.27647E-04	-2.65141E+03	-2.20259E-08
1- 13	12.000	1.65389E-02	-4.62725E-04	-2.86890E+03	-2.14483E-08
1- 14	13.000	1.60589E-02	-4.94835E-04	-3.08028E+03	-2.08258E-08
1- 15	14.000	1.55453E-02	-5.29906E-04	-3.28542E+03	-2.01597E-08
1- 16	15.000	1.49991E-02	-5.61867E-04	-3.48358E+03	-1.94514E-08
1- 17	16.000	1.44216E-02	-5.92653E-04	-3.67445E+03	-1.87024E-08
1- 18	17.000	1.38138E-02	-6.22198E-04	-3.85763E+03	-1.79143E-08
1- 19	18.000	1.31772E-02	-6.50442E-04	-4.03274E+03	-1.70887E-08
1- 20	19.000	1.25130E-02	-6.77324E-04	-4.19941E+03	-1.62273E-08
1- 21	20.000	1.18226E-02	-7.02788E-04	-4.35729E+03	-1.53319E-08
1- 22	21.000	1.11074E-02	-7.26782E-04	-4.50605E+03	-1.44045E-08
1- 23	22.000	1.03690E-02	-7.49255E-04	-4.64538E+03	-1.34469E-08
1- 24	23.000	9.60891E-03	-7.70160E-04	-4.77499E+03	-1.24612E-08

TABLE II. MODE 3
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
1- 25	24.000	8.82870E-03	-7.89454E-04	-4.89461E+03	-1.14494E-08
1- 26	25.000	8.03001E-03	-8.07095E-04	-5.00359E+03	-1.04136E-08
1- 27	26.000	7.21452E-03	-8.23047E-04	-5.10289E+03	-9.35606E-09
1- 28	27.000	6.38393E-03	-8.37276E-04	-5.19111E+03	-8.27891E-09
1- 29	28.000	5.53997E-03	-8.49754E-04	-5.26847E+03	-7.19444E-09
1- 30	29.000	4.68443E-03	-8.60453E-04	-5.33481E+03	-6.07494E-09
1- 31	30.000	3.81907E-03	-8.69351E-04	-5.38998E+03	-4.95272E-09
1- 32	31.000	2.94573E-03	-8.76430E-04	-5.43387E+03	-3.82013E-09
1- 33	32.000	2.06622E-03	-8.81675E-04	-5.46629E+03	-2.67955E-09
1- 34	33.000	1.18238E-03	-8.85075E-04	-5.48746E+03	-1.53335E-09
1- 35	34.000	2.95069E-04	-8.86622E-04	-5.49706E+03	-3.83953E-10
1- 36	35.000	-5.90843E-04	-8.83315E-04	-5.49515E+03	7.66254E-10
1- 37	36.000	-1.47656E-03	-8.84152E-04	-5.48174E+03	1.91486E-09
1- 38	37.000	-2.35917E-03	-8.80139E-04	-5.45686E+03	3.05946E-09
1- 39	38.000	-3.23684E-03	-8.74283E-04	-5.42056E+03	4.19765E-09
1- 40	39.000	-4.10774E-03	-8.65599E-04	-5.37291E+03	5.32707E-09
1- 41	40.000	-4.97004E-03	-8.57100E-04	-5.31402E+03	6.44533E-09
2- 1	40.000	-4.97004E-03	-1.29683E-04	-8.04036E+02	6.44533E-09
2- 2	41.000	-5.09452E-03	-1.19152E-04	-7.38742E+02	6.60677E-09
2- 3	42.000	-5.20835E-03	-1.08371E-04	-6.71901E+02	6.75438E-09
2- 4	43.000	-5.31127E-03	-9.73637E-05	-6.03655E+02	6.88784E-09
2- 5	44.000	-5.40308E-03	-8.61524E-05	-5.34145E+02	7.00692E-09
2- 6	45.000	-5.48258E-03	-7.47608E-05	-4.63517E+02	7.11132E-09
2- 7	46.000	-5.55261E-03	-6.32128E-05	-3.91919E+02	7.20083E-09
2- 8	47.000	-5.61001E-03	-5.15324E-05	-3.19501E+02	7.27528E-09
2- 9	48.000	-5.65568E-03	-3.97442E-05	-2.46414E+02	7.33450E-09
2- 10	49.000	-5.68951E-03	-2.78728E-05	-1.72811E+02	7.37837E-09
2- 11	50.000	-5.71143E-03	-1.59431E-05	-9.88470E+01	7.40680E-09
2- 12	51.000	-5.72140E-03	-3.97995E-06	-2.46757E+01	7.41973E-09
2- 13	52.000	-5.71940E-03	7.99152E-06	4.95474E+01	7.41713E-09
2- 14	53.000	-5.70542E-03	1.99463E-05	1.23667E+02	7.39901E-09
2- 15	54.000	-5.67951E-03	3.18593E-05	1.97528E+02	7.36540E-09
2- 16	55.000	-5.64171E-03	4.37057E-05	2.70975E+02	7.31638E-09
2- 17	56.000	-5.59211E-03	5.54606E-05	3.43856E+02	7.25205E-09

TABLE II, MODE 3
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
2- 18	57.000	-5.53080E-03	6.75994E-05	4.16016E+02	7.17254E-09
2- 19	58.000	-5.45791E-03	7.85979E-05	4.87307E+02	7.07803E-09
2- 20	59.000	-5.37361E-03	8.99318E-05	5.57577E+02	6.96870E-09
2- 21	60.000	-5.27806E-03	1.01078E-04	6.26681E+02	6.84478E-09
2- 22	61.000	-5.17146E-03	1.12012E-04	6.94474E+02	6.70654E-09
2- 23	62.000	-5.05404E-03	1.22712E-04	7.60813E+02	6.55427E-09
2- 24	63.000	-4.92604E-03	1.33155E-04	8.25560E+02	6.38827E-09
2- 25	64.000	-4.78772E-03	1.43219E-04	8.88579E+02	6.20891E-09
2- 26	65.000	-4.63941E-03	1.55184E-04	9.49739E+02	6.01656E-09
2- 27	66.000	-4.48127E-03	1.62728E-04	1.00891E+03	5.81161E-09
2- 28	67.000	-4.31396E-03	1.71921E-04	1.06597E+03	5.59450E-09
2- 29	68.000	-4.13752E-03	1.80775E-04	1.12080E+03	5.36568E-09
2- 30	69.000	-3.95241E-03	1.89240E-04	1.17329E+03	5.12564E-09
2- 31	70.000	-3.75904E-03	1.97209E-04	1.22332E+03	4.87486E-09
2- 32	71.000	-3.55780E-03	2.04965E-04	1.27079E+03	4.61389E-09
2- 33	72.000	-3.34911E-03	2.12193E-04	1.31560E+03	4.34325E-09
2- 34	73.000	-3.13342E-03	2.18976E-04	1.35765E+03	4.06353E-09
2- 35	74.000	-2.91116E-03	2.25301E-04	1.39687E+03	3.77531E-09
2- 36	75.000	-2.68282E-03	2.31155E-04	1.43316E+03	3.47918E-09
2- 37	76.000	-2.44885E-03	2.36525E-04	1.46645E+03	3.17577E-09
2- 38	77.000	-2.20977E-03	2.41399E-04	1.49668E+03	2.86572E-09
2- 39	78.000	-1.96624E-03	2.45769E-04	1.52377E+03	2.54966E-09
2- 40	79.000	-1.71824E-03	2.49624E-04	1.54767E+03	2.22827E-09
2- 41	80.000	-1.45682E-03	2.52957E-04	1.56834E+03	1.90222E-09
2- 42	81.000	-1.21232E-03	2.55761E-04	1.58572E+03	1.57219E-09
2- 43	82.000	-9.55255E-04	2.58029E-04	1.59978E+03	1.23886E-09
2- 44	83.000	-6.96267E-04	2.59758E-04	1.61050E+03	9.02945E-10
2- 45	84.000	-4.35781E-04	2.60942E-04	1.61784E+03	5.65137E-10
2- 46	85.000	-1.74383E-04	2.61581E-04	1.62180E+03	2.26147E-10
2- 47	86.000	8.73798E-05	2.61672E-04	1.62237E+03	-1.13317E-10
2- 48	87.000	3.48960E-04	2.61216E-04	1.61954E+03	-4.52545E-10
2- 49	88.000	6.09811E-04	2.60212E-04	1.61332E+03	-7.90826E-10
2- 50	89.000	8.69388E-04	2.58665E-04	1.60372E+03	-1.12745E-09
2- 51	90.000	1.12714E-03	2.56576E-04	1.59077E+03	-1.46172E-09

TABLE II, MODE 3

TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TOQUE	TORQUE-PRIME
2- 52	91.000	1.38254E-03	2.83950E-04	1.57449E+03	-1.79293E-09
2- 53	92.000	1.63504E-03	2.50792E-04	1.55491E+03	-2.12038E-09
2- 54	93.000	1.88412E-03	2.47110E-04	1.53208E+03	-2.44340E-09
2- 55	94.000	2.12926E-03	2.42911E-04	1.50605E+03	-2.76131E-09
2- 56	95.000	2.36965E-03	2.38203E-04	1.47686E+03	-3.07244E-09
2- 57	96.000	2.60567E-03	2.32997E-04	1.44645E+03	-3.37913E-09
2- 58	97.000	2.83594E-03	2.27303E-04	1.40928E+03	-3.67776E-09
2- 59	98.000	3.06028E-03	2.21133E-04	1.37103E+03	-3.96869E-09
2- 60	99.000	3.27821E-03	2.14501E-04	1.32991E+03	-4.25131E-09
2- 61	100.000	3.48929E-03	2.07420E-04	1.28600E+03	-4.52504E-09
2- 62	101.000	3.69206E-03	1.99904E-04	1.23941E+03	-4.78929E-09
2- 63	102.000	3.88910F-03	1.91970F-04	1.19022E+03	-5.04353E-09
2- 64	103.000	4.07700F-03	1.83635E-04	1.13954E+03	-5.28721E-09
2- 65	104.000	4.25637E-03	1.74915F-04	1.08467E+03	-5.51982E-09
2- 66	105.000	4.42484E-03	1.65829E-04	1.02814E+03	-5.74089E-09
2- 67	106.000	4.58804E-03	1.56396E-04	9.69658E+02	-5.94994E-09
2- 68	107.000	4.73967E-03	1.46636E-04	9.09144E+02	-6.14653E-09
2- 69	108.000	4.88131E-03	1.35669E-04	8.46728E+02	-6.33027E-09
2- 70	109.000	5.01278E-03	1.26216E-04	7.82540E+02	-6.50076E-09
2- 71	110.000	5.13375E-03	1.15599E-04	7.16714E+02	-6.65764E-09
2- 72	111.000	5.24398E-03	1.04740E-04	6.49388E+02	-6.80059E-09
2- 73	112.000	5.34324E-03	9.36617E-05	5.80703E+02	-6.92931E-09
2- 74	113.000	5.43131E-03	8.23875E-05	5.10802E+02	-7.04353E-09
2- 75	114.000	5.50802E-03	7.09408E-05	4.39832E+02	-7.14300E-09
2- 76	115.000	5.57220E-03	5.93456E-05	3.67942E+02	-7.22753E-09
2- 77	116.000	5.62671E-03	4.75262E-05	2.95283E+02	-7.29693E-09
2- 78	117.000	5.66844E-03	3.58072E-05	2.22908E+02	-7.35107E-09
2- 79	118.000	5.69823E-03	2.39132E-05	1.48262E+02	-7.38981E-09
2- 80	119.000	5.71629E-03	1.19691E-05	7.42086E+01	-7.41310E-09
2- 81	120.000	5.72228E-03	-1.45815E-18	-9.04052E-12	-7.42087E-09
3- 1	40.000	-4.97004E-03	-5.88011E-02	-4.50999E+03	1.31836E-10
3- 2	41.000	-6.37626E-02	-5.85822E-02	-4.50087E+03	1.69138E-09
3- 3	42.000	-1.22335E-01	-5.83602E-02	-4.47617E+03	3.24507E-09
3- 4	43.000	-1.80483E-01	-5.78362E-02	-4.43598E+03	4.78754E-09

TABLE II, MODE 3
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
3- 5	44.000	-2.38007E-01	-5.71120E-02	-4.38043E+03	6.31343E-09
3- 6	45.000	-2.94708E-01	-5.41901E-02	-4.20972E+03	7.81748E-09
3- 7	46.000	-3.50288E-01	-5.50737E-02	-4.22410E+03	9.29448E-09
3- 8	47.000	-4.04856E-01	-5.37667E-02	-4.12385E+03	1.07393E-08
3- 9	48.000	-4.57923E-01	-5.22735E-02	-4.00932E+03	1.21470E-08
3- 10	49.000	-5.09404E-01	-5.05994E-02	-3.88092E+03	1.35126E-08
3- 11	50.000	-5.59123E-01	-4.87501E-02	-3.73909E+03	1.48314E-08
3- 12	51.000	-6.06906E-01	-4.67321E-02	-3.58431E+03	1.60989E-08
3- 13	52.000	-6.52589E-01	-4.45523E-02	-3.41712E+03	1.73107E-08
3- 14	53.000	-6.96012E-01	-4.22183E-02	-3.23810E+03	1.84626E-08
3- 15	54.000	-7.37027E-01	-3.97382E-02	-3.04788E+03	1.95505E-08
3- 16	55.000	-7.74911E-01	-3.71204E-02	-2.84710E+03	2.05708E-08
3- 17	56.000	-8.11270E-01	-3.43742E-02	-2.63647E+03	2.15199E-08
3- 18	57.000	-8.44242E-01	-3.15090E-02	-2.41671E+03	2.23945E-08
3- 19	58.000	-8.74291E-01	-2.85347E-02	-2.18859E+03	2.31916E-08
3- 20	59.000	-9.01314E-01	-2.54617E-02	-1.95289E+03	2.39084E-08
3- 21	60.000	-9.25217E-01	-2.23005E-02	-1.71042E+03	2.45425E-08
3- 22	61.000	-9.45917E-01	-1.90621E-02	-1.46204E+03	2.50916E-08
3- 23	62.000	-9.63344E-01	-1.57577E-02	-1.20860E+03	2.55539E-08
3- 24	63.000	-9.77435E-01	-1.23987E-02	-9.50968E+02	2.59277E-08
3- 25	64.000	-9.88144E-01	-8.99682E-03	-6.90047E+02	2.62117E-08
3- 26	65.000	-9.95432E-01	-5.56378E-03	-4.26737E+02	2.64050E-08
3- 27	66.000	-9.99274E-01	-2.111148E-03	-1.61948E+02	2.65070E-08
3- 28	67.000	-9.99458E-01	1.34815E-03	1.03401E+02	2.65171E-08
3- 29	68.000	-9.96581E-01	4.80311E-03	3.68394E+02	2.64355E-08
3- 30	69.000	-9.90055E-01	8.24147E-03	6.32112E+02	2.62624E-08
3- 31	70.000	-9.80101E-01	1.16513E-02	8.93643E+02	2.59984E-08
3- 32	71.000	-9.66755E-01	1.50208E-02	1.15208E+03	2.56444E-08
3- 33	72.000	-9.50062E-01	1.83383E-02	1.40653E+03	2.52016E-08
3- 34	73.000	-9.30C81E-01	2.15924E-02	1.65612E+03	2.46715E-08
3- 35	74.000	-9.06881E-01	2.47717E-02	1.89997E+03	2.40561E-08
3- 36	75.000	-8.80541E-01	2.78653E-02	2.13724E+03	2.33574E-08
3- 37	76.000	-8.51153E-01	3.08625E-02	2.36712E+03	2.25779E-08
3- 38	77.000	-8.18818E-01	3.37528E-02	2.58881E+03	2.17201E-08

TABLE II, MODE 3
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
3- 39	78.000	-7.83649E-01	3.65263E-02	2.80153E+03	2.07873E-08
3- 40	79.000	-7.45748E-01	2.91734E-02	3.00454E+03	1.97824E-08
3- 41	80.000	-7.05205E-01	4.16849E-02	3.19719E+03	1.07091E-08
3- 42	81.000	-6.62401E-01	4.40521E-02	3.37875E+03	1.75710E-08
3- 43	82.000	-6.17203E-01	4.62668E-02	3.54862E+03	1.63721E-08
3- 44	83.000	-5.69849E-01	4.83214E-02	3.70620E+03	1.51165E-08
3- 45	84.000	-5.20562E-01	5.02087E-02	3.85096E+03	1.38085E-08
3- 46	85.000	-4.69453E-01	5.19222E-02	3.98238E+03	1.24520E-08
3- 47	86.000	-4.16719E-01	5.34560E-02	4.10003E+03	1.10540E-08
3- 48	87.000	-3.62562E-01	5.48048E-02	4.20348E+03	9.61688E-09
3- 49	88.000	-3.07111E-01	5.59679E-02	4.29238E+03	8.14649E-09
3- 50	89.000	-2.50616E-01	5.69293E-02	4.36642E+03	6.64789E-09
3- 51	90.000	-1.93253E-01	5.76976E-02	4.42535E+03	5.12627E-09
3- 52	91.000	-1.35221E-01	5.82662E-02	4.46896E+03	3.58691E-09
3- 53	92.000	-7.47214E-02	5.86321E-02	4.49710E+03	2.03513E-09
3- 54	93.000	-1.79575E-02	5.87970E-02	4.50967E+03	4.76296E-10
3- 55	94.000	4.08724E-02	5.87574E-02	4.50564E+03	-1.08419E-09
3- 56	95.000	9.95591E-02	5.85145E-02	4.48800E+03	-2.64093E-09
3- 57	96.000	1.57901E-01	5.80690E-02	4.45382E+03	-4.18853E-09
3- 58	97.000	2.15697E-01	5.74224E-02	4.40424E+03	-5.72164E-09
3- 59	98.000	2.72747E-01	5.65772E-02	4.33941E+03	-7.23494E-09
3- 60	99.000	3.28862E-01	5.55360E-02	4.25956E+03	-8.72321E-09
3- 61	100.000	3.83820E-01	5.43027E-02	4.16496E+03	-1.01813E-08
3- 62	101.000	4.37459E-01	5.28813E-02	4.05595E+03	-1.16041E-08
3- 63	102.000	4.89584E-01	5.12770E-02	3.93289E+03	-1.29868E-08
3- 64	103.000	5.40014E-01	4.94951E-02	3.79622E+03	-1.43245E-08
3- 65	104.000	5.88575E-01	4.75418E-02	3.64641E+03	-1.56127E-08
3- 66	105.000	6.35099E-01	4.54241E-02	3.48398E+03	-1.68468E-08
3- 67	106.000	6.79425E-01	4.31490E-02	3.30949E+03	-1.80226E-08
3- 68	107.000	7.21399E-01	4.07246E-02	3.12354E+03	-1.91360E-08
3- 69	108.000	7.60876E-01	3.81593E-02	2.92678E+03	-2.01832E-08
3- 70	109.000	7.97720E-01	3.54618E-02	2.71988E+03	-2.11605E-08
3- 71	110.000	8.31802E-01	3.26416E-02	2.50358E+03	-2.20646E-08
3- 72	111.000	8.63005E-01	2.97084E-02	2.27860E+03	-2.28923E-08

TABLE II, MODE 3

TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
3- 73	112.000	8.91221E-01	2.66723E-02	2.04574E+03	-2.36407E-08
3- 74	113.000	9.16253E-01	2.35439E-02	1.80579E+03	-2.43074E-08
3- 75	114.000	9.38312E-01	2.03340E-02	1.55960E+03	-2.48899E-08
3- 76	115.000	9.57023E-01	1.70537E-02	1.30800E+03	-2.53862E-08
3- 77	116.000	9.72622E-01	1.37144E-02	1.05188E+03	-2.57947E-08
3- 78	117.000	9.84455E-01	1.03275E-02	7.92112E+02	-2.61139E-08
3- 79	118.000	9.93080E-01	6.90496E-03	5.29604E+02	-2.63427E-08
3- 80	119.000	9.98268E-01	3.45847E-03	2.65261E+02	-2.64803E-08
3- 81	120.000	1.000000E+00	1.01032E-15	7.74903E-11	-2.65262E-08

143

TOTAL NO. STATIONS = 203

TABLE II
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE
 $T = 0.00$ SEC
MODE 4

FREQUENCY CYCLES PER SECOND 1399.407

FREQUENCY RADIANS PER SECOND 8792.7318

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
1- 1	0.000	9.914C8E-01	0.	0.	-1.68625E-06
1- 2	1.000	9.90C49E-01	-2.71976E-03	-1.68625E+04	-1.68394E-06
1- 3	2.000	9.85971E-01	-5.43207E-03	-3.38788E+04	-1.67701E-06
1- 4	3.000	9.79186E-01	-8.12946E-03	-5.04027E+04	-1.665547E-06
1- 5	4.000	9.69714E-01	-1.08045E-02	-6.69882E+04	-1.64935E-06
1- 6	5.000	9.57E79E-01	-1.34E00E-02	-8.33898E+04	-1.62871E-06
1- 7	6.000	9.42816E-01	-1.60585E-02	-9.95626E+04	-1.60260E-06
1- 8	7.000	9.25464E-01	-1.86229E-02	-1.1E482E+05	-1.57409E-06
1- 9	8.000	9.05E72E-01	-2.11262E-02	-1.31045E+05	-1.54026E-06
1- 10	9.000	8.83193E-01	-2.3E915E-02	-1.46267E+05	-1.50219E-06
1- 11	10.000	8.58390E-01	-2.59821E-02	-1.61089E+05	-1.46001E-06
1- 12	11.000	8.31231E-01	-2.83013E-02	-1.75468E+05	-1.41381E-06
1- 13	12.000	8.01789E-01	-3.05428E-02	-1.89367E+05	-1.36374E-06
1- 14	13.000	7.70147E-01	-3.27005E-02	-2.02742E+05	-1.30992E-06
1- 15	14.000	7.36390E-01	-3.47684E-02	-2.15564E+05	-1.25250E-06
1- 16	15.000	7.00611E-01	-3.67409E-02	-2.27793E+05	-1.19165E-06
1- 17	16.000	6.62910E-01	-3.86125E-02	-2.39397E+05	-1.12752E-06
1- 18	17.000	6.23388E-01	-4.03781E-02	-2.50344E+05	-1.06030E-06
1- 19	18.000	5.82155E-01	-4.20329E-02	-2.60604E+05	-9.90167E-07
1- 20	19.000	5.39323E-01	-4.35723E-02	-2.70148E+05	-9.17317E-07
1- 21	20.000	4.95011E-01	-4.49920E-02	-2.78951E+05	-8.41948E-07
1- 22	21.000	4.49340E-01	-4.62883E-02	-2.86987E+05	-7.64268E-07
1- 23	22.000	4.02435E-01	-4.74575E-02	-2.94236E+05	-6.84489E-07
1- 24	23.000	3.54426E-01	-4.84964E-02	-3.00678E+05	-6.02831E-07

TABLE II, MODE 4
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
1- 25	24.000	3.05442E-01	-4.94022E-02	-3.06294E+05	-5.19519E-07
1- 26	25.000	2.55622E-01	-5.01724E-02	-3.11069E+05	-4.34779E-07
1- 27	26.000	2.05099E-01	-5.08048E-02	-3.14990E+05	-3.48847E-07
1- 28	27.000	1.54013E-01	-5.12978E-02	-3.18046E+05	-2.61956E-07
1- 29	28.000	1.02504E-01	-5.16499E-02	-3.20230E+05	-1.74346E-07
1- 30	29.000	5.07135E-02	-5.18603E-02	-3.21534E+05	-8.62570E-08
1- 31	30.000	-1.21631E-03	-5.19283E-02	-3.21955E+05	2.06878E-09
1- 32	31.000	-5.31429E-02	-5.18537E-02	-3.21493E+05	9.03890E-08
1- 33	32.000	-1.04924E-01	-5.16368E-02	-3.20148E+05	1.78461E-07
1- 34	33.000	-1.54417E-01	-5.12781E-02	-3.17924E+05	2.65044E-07
1- 35	34.000	-2.07480E-01	-5.07787E-02	-3.14828E+05	3.52896E-07
1- 36	35.000	-2.57974E-01	-5.01398E-02	-3.10867E+05	4.38780E-07
1- 37	36.000	-3.07760E-01	-4.93634E-02	-3.06053E+05	5.23459E-07
1- 38	37.000	-3.56701E-01	-4.84514E-02	-3.00398E+05	6.06702E-07
1- 39	38.000	-4.04664E-01	-4.74064E-02	-2.93919E+05	6.88279E-07
1- 40	39.000	-4.51515E-01	-4.62212E-02	-2.86632E+05	7.67967E-07
1- 41	40.000	-4.97127E-01	-4.49291E-02	-2.78561E+05	8.45546E-07
2- 1	40.000	-4.97127E-01	-4.54417E-02	-2.81730E+05	8.45546E-07
2- 2	41.000	-5.41887E-01	-4.40154E-02	-2.72897E+05	9.21677E-07
2- 3	42.000	-5.85159E-01	-4.24686E-02	-2.63305E+05	9.95277E-07
2- 4	43.000	-6.26875E-01	-4.08051E-02	-2.52992E+05	1.06615E-06
2- 5	44.000	-6.66770E-01	-3.90295E-02	-2.41983E+05	1.13409E-06
2- 6	45.000	-7.04885E-01	-3.71468E-02	-2.30310E+05	1.19892E-06
2- 7	46.000	-7.41065E-01	-3.51621E-02	-2.18005E+05	1.26045E-06
2- 8	47.000	-7.75211E-01	-3.30809E-02	-2.05102E+05	1.31853E-06
2- 9	48.000	-8.07228E-01	-3.09089E-02	-1.91635E+05	1.37299E-06
2- 10	49.000	-8.37030E-01	-2.86520E-02	-1.77642E+05	1.42368E-06
2- 11	50.000	-8.64534E-01	-2.63164E-02	-1.62162E+05	1.47046E-06
2- 12	51.000	-8.89664E-01	-2.39086E-02	-1.48233E+05	1.51320E-06
2- 13	52.000	-9.12253E-01	-2.14352E-02	-1.32898E+05	1.55179E-06
2- 14	53.000	-9.32536E-01	-1.89029E-02	-1.17198E+05	1.58612E-06
2- 15	54.000	-9.50160E-01	-1.63187E-02	-1.01174E+05	1.61610E-06
2- 16	55.000	-9.65175E-01	-1.36897E-02	-8.48763E+04	1.64163E-06
2- 17	56.000	-9.77541E-01	-1.10231E-02	-6.83435E+04	1.66267E-06

TABLE II, MODE 4
TORSTIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
2- 18	57.000	-9.87224E-01	-8.32630E-03	-5.16231E+04	1.67914E-06
2- 19	58.000	-9.94196E-01	-5.60660E-03	-3.47609E+04	1.69099E-06
2- 20	59.000	-9.98439E-01	-2.87149E-03	-1.78033E+04	1.69821E-06
2- 21	60.000	-9.99941E-01	-1.28506E-04	-7.96736E+02	1.70077E-06
2- 22	61.000	-9.98697E-01	2.61484E-03	1.62120E+04	1.69865E-06
2- 23	62.000	-9.94713E-01	5.35101E-03	3.31763E+04	1.69187E-06
2- 24	63.000	-9.87957E-01	8.07250E-03	5.00495E+04	1.68045E-06
2- 25	64.000	-9.78570E-01	1.07718E-02	6.67854E+04	1.66442E-06
2- 26	65.000	-9.66456E-01	1.34416E-02	8.33380E+04	1.64281E-06
2- 27	66.000	-9.51688E-01	1.60745E-02	9.96618E+04	1.61870E-06
2- 28	57.000	-9.34308E-01	1.85632E-02	1.15712E+05	1.58913E-06
2- 29	68.000	-9.14364E-01	2.12007E-02	1.31445E+05	1.55521E-06
2- 30	69.000	-8.91909E-01	2.36801E-02	1.46814E+05	1.51702E-06
2- 31	70.000	-8.67005E-01	2.60944E-02	1.61785E+05	1.47466E-06
2- 32	71.000	-8.39722E-01	2.84271E-02	1.76310E+05	1.42825E-06
2- 33	72.000	-8.10133E-01	3.07017E-02	1.90351E+05	1.37793E-06
2- 34	73.000	-7.78220E-01	3.28821E-02	2.03869E+05	1.32382E-06
2- 35	74.000	-7.44370E-01	3.49722E-02	2.16827E+05	1.26607E-06
2- 36	75.000	-7.08377E-01	3.69662E-02	2.29191E+05	1.20485E-06
2- 37	76.000	-6.70439E-01	3.88588E-02	2.40925E+05	1.14033E-06
2- 38	77.000	-6.30650E-01	4.06448E-02	2.51998E+05	1.07267E-06
2- 39	78.000	-5.89151E-01	4.23191E-02	2.62379E+05	1.00207E-06
2- 40	79.000	-5.46023E-01	4.38773E-02	2.72029E+05	9.28713E-07
2- 41	80.000	-5.01297E-01	4.53151E-02	2.80952E+05	8.52810E-07
2- 42	81.000	-4.55394E-01	4.66284E-02	2.88906E+05	7.74565E-07
2- 43	82.000	-4.08141E-01	4.78138E-02	2.96445E+05	6.94194E-07
2- 44	83.000	-3.59748E-01	4.88678E-02	3.02981E+05	6.11917E-07
2- 45	84.000	-3.10426E-01	4.97878E-02	3.08684E+05	5.27960E-07
2- 46	85.000	-2.60193E-01	5.05710E-02	3.13540E+05	4.42553E-07
2- 47	86.000	-2.09265E-01	5.12155E-02	3.17526E+05	3.55932E-07
2- 48	87.000	-1.57762E-01	5.17193E-02	3.20560E+05	2.68333E-07
2- 49	88.000	-1.05827E-01	5.20811E-02	3.22903E+05	1.79997E-07
2- 50	89.000	-5.36003E-02	5.23000E-02	3.24260E+05	9.11670E-08
2- 51	90.000	-1.22678E-03	5.23753E-02	3.24727E+05	2.08659E-09

TABLE II, MODE 4

TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TOQUE	TORQUE-PRIME
2- 52	91.000	5.11502E-02	5.23068E-02	3.24302E+05	-8.69997E-08
2- 53	92.000	1.03387E-01	5.20948E-02	3.22988E+05	-1.75847E-07
2- 54	92.000	1.55240E-01	5.17397E-02	3.20786E+05	-2.64212E-07
2- 55	94.000	2.06867E-01	5.12426E-02	3.17704E+05	-3.51852E-07
2- 56	95.000	2.57825E-01	5.06048E-02	3.13750E+05	-4.38527E-07
2- 57	96.000	3.08076E-01	4.98281E-02	3.08934E+05	-5.23997E-07
2- 58	97.000	3.57482E-01	4.89146E-02	3.03270E+05	-6.09029E-07
2- 59	98.000	4.05906E-01	4.78658E-02	2.96774E+05	-6.90392E-07
2- 60	99.000	4.53216E-01	4.66876E-02	2.89462E+05	-7.70860E-07
2- 61	100.000	4.99282E-01	4.53802E-02	2.81357E+05	-8.49212E-07
2- 62	101.000	5.43979E-01	4.39483E-02	2.72479E+05	-9.25233E-07
2- 63	102.000	5.87180E-01	4.23957E-02	2.62853E+05	-9.98714E-07
2- 64	103.000	6.28770E-01	4.07267E-02	2.52506E+05	-1.06945E-06
2- 65	104.000	6.68634E-01	3.89459E-02	2.41465E+05	-1.13726E-06
2- 66	105.000	7.04663E-01	3.70582E-02	2.29761E+05	-1.20194E-06
2- 67	106.000	7.42782E-01	3.50688E-02	2.17426E+05	-1.26332E-06
2- 68	107.000	7.76802E-01	3.29830E-02	2.04495E+05	-1.32124E-06
2- 69	108.000	8.08719E-01	3.08068E-02	1.91002E+05	-1.37552E-06
2- 70	109.000	8.38417E-01	2.85459E-02	1.76985E+05	-1.42604E-06
2- 71	110.000	8.65813E-01	2.62067E-02	1.62482E+05	-1.47263E-06
2- 72	111.000	8.90832E-01	2.37956E-02	1.47532E+05	-1.51519E-06
2- 73	112.000	9.13405E-01	2.13191E-02	1.32178E+05	-1.55358E-06
2- 74	113.000	9.33472E-01	1.87841E-02	1.16461E+05	-1.58771E-06
2- 75	114.000	9.50975E-01	1.61975E-02	1.00424E+05	-1.61748E-06
2- 76	115.000	9.65888E-01	1.35664E-02	8.41117E+04	-1.64281E-06
2- 77	116.000	9.78110E-01	1.098981E-02	6.75582E+04	-1.66363E-06
2- 78	117.000	9.87666E-01	8.19987E-03	5.08392E+04	-1.67989E-06
2- 79	118.000	9.94511E-01	5.47912E-03	3.29705E+04	-1.69153E-06
2- 80	119.000	9.98626E-01	2.74333E-03	1.70086E+04	-1.69853E-06
2- 81	120.000	1.00000E+00	1.39183E-15	8.62937E-09	-1.70087E-06
3- 1	40.000	-4.97127E-01	4.14331E-02	3.17788E+03	1.72953E-08
3- 2	41.000	-4.54566E-01	4.35941E-02	3.34362E+03	1.58146E-08
3- 3	42.000	-4.09941E-01	4.55571E-02	3.49418E+03	1.42620E-08
3- 4	43.000	-3.63454E-01	4.73132E-02	3.62888E+03	1.26447E-08

TABLE II, MODE 4
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
3- 5	44.000	-3.15317E-01	4.88544E-02	3.74710E+03	1.09700E-08
3- 6	45.000	-2.65747E-01	5.01740E-02	3.84830E+03	9.24546E-09
3- 7	46.000	-2.14970E-01	5.12657E-02	3.93202E+03	7.47892E-09
3- 8	47.000	-1.63217E-01	5.21245E-02	3.99790E+03	5.67840E-09
3- 9	48.000	-1.10723E-01	5.27466E-02	4.04561E+03	3.85209E-09
3- 10	49.000	-5.77248E-02	5.31292E-02	4.07496E+03	2.00827E-09
3- 11	50.000	-4.46464E-03	5.32706E-02	4.08580E+03	1.55327E-10
3- 12	51.000	4.88160E-02	5.31700E-02	4.07809E+03	-1.69833E-09
3- 13	52.000	1.01975E-01	5.28280E-02	4.05185E+03	-3.54429E-09
3- 14	53.000	1.54472E-01	5.22461E-02	4.00722E+03	-5.37416E-09
3- 15	54.000	2.06368E-01	5.14269E-02	3.94439E+03	-7.17964E-09
3- 16	55.000	2.57327E-01	5.03742E-02	3.86365E+03	-8.95252E-09
3- 17	56.000	3.07117E-01	4.90927E-02	3.76535E+03	-1.06848E-08
3- 18	57.000	3.55514E-01	4.75883E-02	3.64997E+03	-1.23685E-08
3- 19	58.000	4.02295E-01	4.58677E-02	3.51801E+03	-1.39960E-08
3- 20	59.000	4.47251E-01	4.39389E-02	3.37007E+03	-1.55601E-08
3- 21	60.000	4.90175E-01	4.18105E-02	3.20583E+03	-1.70574E-08
3- 22	61.000	5.30874E-01	3.94923E-02	3.02902E+03	-1.84694E-08
3- 23	62.000	5.69143E-01	3.69947E-02	2.82744E+03	-1.98014E-08
3- 24	63.000	6.04846E-01	3.43291E-02	2.63301E+03	-2.10436E-08
3- 25	64.000	6.37824E-01	3.15076E-02	2.41660E+03	-2.21902E-08
3- 26	65.000	6.67885E-01	2.85430E-02	2.18922E+03	-2.32360E-08
3- 27	66.000	6.94913E-01	2.54487E-02	1.95189E+03	-2.41763E-08
3- 28	67.000	7.18784E-01	2.22389E-02	1.70570E+03	-2.50049E-08
3- 29	68.000	7.39394E-01	1.89281E-02	1.45176E+03	-2.57239E-08
3- 30	69.000	7.56645E-01	1.55313E-02	1.19123E+03	-2.63240E-08
3- 31	70.000	7.70451E-01	1.20639E-02	9.25291E+02	-2.68047E-08
3- 32	71.000	7.80777E-01	8.54177E-03	6.55144E+02	-2.71626E-08
3- 33	72.000	7.87548E-01	4.98082E-03	3.82024E+02	-2.73992E-08
3- 34	73.000	7.90743E-01	1.39722E-03	1.07166E+02	-2.75103E-08
3- 35	74.000	7.90347E-01	-2.19273E-03	-1.68180E+02	-2.74965E-08
3- 36	75.000	7.86361E-01	-5.77275E-03	-4.42764E+02	-2.73579E-08
3- 37	76.000	7.78805E-01	-9.32657E-03	-7.15339E+02	-2.70950E-08
3- 38	77.000	7.67712E-01	-1.28381E-02	-9.84666E+02	-2.67091E-08

TABLE II, MODE 4
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
3- 39	78.000	7.53133E-01	-1.62913E-02	-1.24952E+03	-2.62018E-08
3- 40	79.000	7.35134E-01	-1.96705E-02	-1.50871E+03	-2.55756E-08
3- 41	80.000	7.13795E-01	-2.29604E-02	-1.76104E+03	-2.48333E-08
3- 42	81.000	6.89217E-01	-2.61461E-02	-2.00538E+03	-2.39782E-08
3- 43	82.000	6.61507E-01	-2.92131E-02	-2.24062E+03	-2.30141E-08
3- 44	83.000	6.30794E-01	-3.21474E-02	-2.46567E+03	-2.19456E-08
3- 45	84.000	5.97216E-01	-3.49358E-02	-2.67954E+03	-2.07774E-08
3- 46	85.000	5.60924E-01	-3.75655E-02	-2.88124E+03	-1.99149E-08
3- 47	86.000	5.22088E-01	-4.00246E-02	-3.06985E+03	-1.81637E-08
3- 48	87.000	4.80879E-01	-4.23020E-02	-3.24452E+03	-1.67300E-08
3- 49	88.000	4.37487E-01	-4.43874E-02	-3.40447E+03	-1.52204E-08
3- 50	89.000	3.92107E-01	-4.62711E-02	-3.54895E+03	-1.36416E-08
3- 51	90.000	3.44947E-01	-4.79448E-02	-3.67731E+03	-1.20009E-08
3- 52	91.000	2.96220E-01	-4.94007E-02	-3.78898E+03	-1.03056E-08
3- 53	92.000	2.46147E-01	-5.06322E-02	-3.88347E+03	-8.56347E-09
3- 54	93.000	1.94955E-01	-5.16340E-02	-3.96027E+03	-6.78262E-09
3- 55	94.000	1.42880E-01	-5.24012E-02	-4.01912E+03	-4.97087E-09
3- 56	95.000	9.01551E-02	-5.29304E-02	-4.04971E+03	-3.13654E-09
3- 57	96.000	3.70202E-02	-5.32193E-02	-4.08187E+03	-1.28795E-09
3- 58	97.000	-1.62831E-02	-5.32666E-02	-4.08549E+03	5.66498E-10
3- 59	98.000	-6.95128E-02	-5.30719E-02	-4.07056E+03	2.41838E-09
3- 60	99.000	-1.22427E-01	-5.26362E-02	-4.02715E+03	4.25929E-09
3- 61	100.000	-1.74785E-01	-5.19615E-02	-4.98540E+03	6.08087E-09
3- 62	101.000	-2.26351E-01	-5.10508E-02	-3.91555E+03	7.87484E-09
3- 63	102.000	-2.76888E-01	-4.99083E-02	-3.82792E+03	9.63307E-09
3- 64	103.000	-3.26169E-01	-4.85392E-02	-3.72291E+03	1.13476E-08
3- 65	104.000	-3.73968E-01	-4.69496E-02	-3.60099E+03	1.30105E-08
3- 66	105.000	-4.20069E-01	-4.51468E-02	-3.46272E+03	1.46144E-08
3- 67	106.000	-4.64264E-01	-4.31390E-02	-3.30872E+03	1.61519E-08
3- 68	107.000	-5.06350E-01	-4.09353E-02	-3.13970E+03	1.76161E-08
3- 69	108.000	-5.46136E-01	-3.85457E-02	-2.95641E+03	1.90003E-08
3- 70	109.000	-5.83443E-01	-3.59810E-02	-2.75970E+03	2.02983E-08
3- 71	110.000	-6.18101E-01	-3.32529E-02	-2.55046E+03	2.15040E-08
3- 72	111.000	-6.49952E-01	-3.03738E-02	-2.32964E+03	2.26121E-08

TABLE II, MODE 4
TORSIONAL MODES OF CYLINDER-SHAFT CONFIGURATION FOR NUMERICAL EXAMPLE

STATION	X	ZETA	ZETA-PRIME	TORQUE	TORQUE-PRIME
3- 73	112.000	-6.78852E-01	-2.73567E-02	-2.09823E+03	2.36176E-08
3- 74	113.000	-7.04669E-01	-2.42154E-02	-1.85720E+03	2.45158E-08
3- 75	114.000	-7.27286E-01	-2.09641E-02	-1.60793E+03	2.53026E-08
3- 76	115.000	-7.46601E-01	-1.76176E-02	-1.35126E+03	2.59746E-08
3- 77	116.000	-7.62525E-01	-1.41911E-02	-1.08945E+03	2.65286E-08
3- 78	117.000	-7.74987E-01	-1.07001E-02	-8.20591E+02	2.69622E-08
3- 79	118.000	-7.83930E-01	-7.15056E-03	-5.49208E+02	2.72733E-08
3- 80	119.000	-7.89312E-01	-3.58844E-03	-2.75230E+02	2.74605E-08
3- 81	120.000	-7.91110E-01	2.33199E-16	1.78861E-11	2.75231E-08

\$NAME

INCHX = 8,
DX = 0.15E+02,
XMTN = 0.0,
PLTZ = 1,
PLTZPR = 1,
NZAPR = 1,
MPLT = 0,
PLTT = 1,
NPLTT = 1,
TMIN = 0.43164351072729E-124,
ZPMIN1 = 1,
DZP1 = 1,
ZPMIN2 = 1,
DZP2 = 1,
ZPMIN3 = 1,
DZP3 = 1,
DTT = 0.22373683230271E-217,
PLTH = 1,
NMZR = 1,
HZRMIN = 1,
DMZR = 0.11279098072731E-47,
PLTZR = 1,
PLTAE = 0,
PLTJG = 1,

NMAEJG = 1,
AJMTN = -0.46812858247686+248,
DAEJG = 0.22452527121716-217,
\$END
TOTAL NUMBER OF PLOTS = 9