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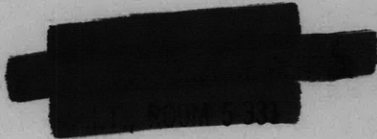


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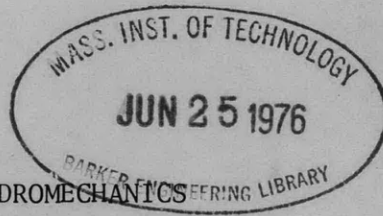


## INTEGRAL METHODS FOR TURBULENT BOUNDARY LAYERS IN PRESSURE GRADIENTS

by

Paul S. Granville

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DEPARTMENT OF HYDROMECHANICS  
RESEARCH AND DEVELOPMENT REPORT

April 1970

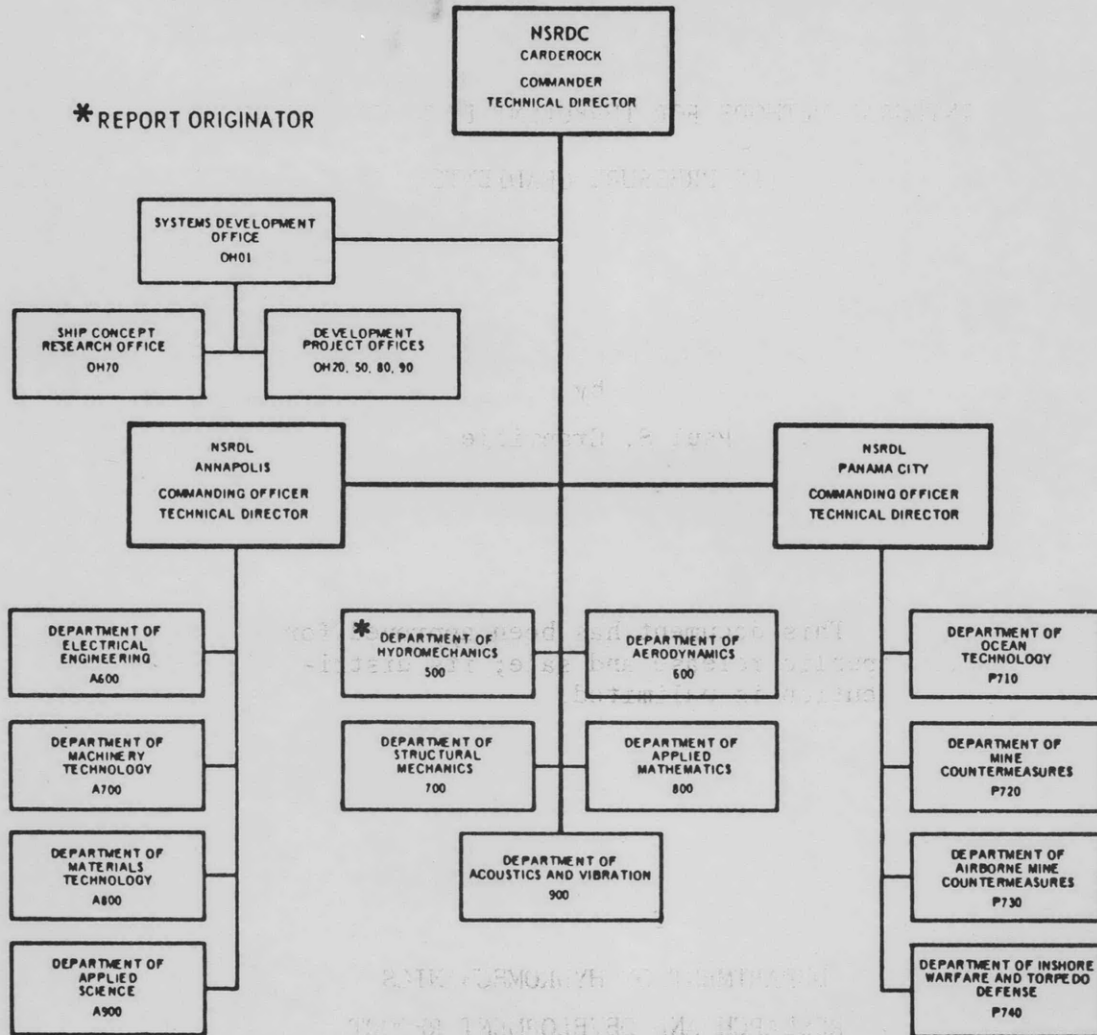
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INTEGRAL METHODS FOR TURBULENT BOUNDARY LAYERS IN PRESSURE GRADIENTS

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## NOTATION

A	Slope of logarithmic velocity law
$\mathcal{A}$	Factor in equation for flat-plate shape parameter, Equation (42)
$B_1, B_2$	Intercepts of logarithmic velocity laws, Equations (6) and (9)
$\mathcal{B}$	Factor in equation for flat-plate shape parameter, Equation (42)
$C_D$	Dissipation integral defined in Equation (89)
$\hat{C}_D$	Dissipation integral defined in Equation (94)
$C_S$	General shear-stress factor given in Equation (82)
$\hat{C}_S$	General shear-stress factor defined in Equation (84)
$C_\tau$	Shear-stress integral defined in Equation (192)
$c_1, c_2$	Constants in Equation (26)
$c_3, c_4$	Constants in Equation (35)
E	Entrainment factor, Equation (115)
e	Subscript denoting equilibrium conditions
G	Rotta's shape parameter defined in Equation (21)
$G^*$	Velocity-defect energy shape parameter defined in Equation (54)
$\hat{G}$	Velocity-defect shape parameter defined in Equation (169)
$\bar{G}$	Velocity-defect shape parameter defined in Equation (172)
H	Shape parameter, $H = \delta^* / \theta$
$H^*$	Energy shape parameter defined in Equation (45)
$\tilde{H}$	Entrainment shape parameter defined in Equation (60)
$\hat{H}$	Shape parameter defined in Equation (140)
$\bar{H}$	Shape parameter defined in Equation (141)
$I_1$	Velocity-defect integral defined in Equation (32)
M	Pressure-gradient coefficient, Equation (82)
m	Relative position in boundary layer, $m = s/\delta$
N	Coefficient in Equation (82)
n	Power-law exponent, Equation (16)
P	Coefficient in Equation (82)
p	Pressure

$R_\theta$	Momentum-thickness Reynolds number, $R_\theta = U\theta/\nu$
$s$	$y$ -position in boundary layer
$U$	Velocity at outer edge of boundary layer
$u$	Mean tangential velocity component in boundary layer
$u'$	Fluctuating tangential velocity
$u_s$	$u$ at position $y = s$
$u_\tau$	Shear velocity, $u_\tau = \sqrt{\tau_w/\rho}$
$u_*$	General shear velocity, $u_* = \sqrt{\tau^*/\rho}$
$v$	Mean normal velocity component
$v'$	Fluctuating normal velocity
$w$	Coles' wake factor
$x$	Streamwise distance
$y$	Normal distance from wall
$o$	Subscript denoting flat-plate conditions
$0.3$	Subscript denoting conditions at $m = 0.3$
$\alpha_1, \alpha_2$	Constants in Equation (28)
$\beta$	Clauser's pressure-gradient parameter defined in Equation (70)
$\gamma$	Velocity ratio, $(u/U)_{y=\theta}$ , Equation (25)
$\delta$	Boundary-layer thickness
$\delta^*$	Displacement thickness
$\zeta$	Factor in Equation (114)
$\theta$	Momentum thickness
$\theta^*$	Energy thickness defined in Equation (44)
$\lambda$	Constant in Equation (202)
$\nu$	Kinematic viscosity
$\rho$	Density
$\tau$	Shear stress in boundary layer
$\tau_s$	$\tau$ at $y = s$
$\tau_w$	Wall shear stress
$\tau^*$	Characteristic shear stress, Equation (3)
$\psi_o$	Factor in Equation (201)





## ABSTRACT

Shape parameter differential equations are developed for turbulent boundary layers in pressure gradients incorporating two-parameter velocity profiles. Energy and entrainment methods are included. Shear stress factors are explicitly developed for equilibrium and quasi-equilibrium conditions.

## ADMINISTRATIVE INFORMATION

This work was funded by the Naval Ordnance Systems Command under Subproject UR 109 01 03.

## INTRODUCTION

The analytical prediction of turbulent boundary layers in pressure gradients has been the subject of intensive investigation not only because of the engineering applications but also because of the difficulties in developing methods suitable for all types of pressure distributions. The fundamental problem of the turbulent boundary layer is common to that of all turbulent flow: the lack of an adequate theory on the mechanics of turbulent motion.

Rotta<sup>1-3</sup> has critically examined the various predictive methods which have appeared in the literature in recent years. The 1968 Stanford Conference<sup>4</sup> tested the ability of current methods to predict existing experimental results from given pressure distributions.

Among the trends which may be ascertained from the Stanford Conference are:

1. The virtual abandonment of traditional  $\theta$ - $H$  formulations ( $\theta$  = momentum thickness,  $H$  = shape parameter) when utilizing two-parameter velocity profiles such as the Coles law of the wake. However results are still given in terms of  $\theta$  and  $H$ .
2. The use of shear stress factors, (e.g., the dissipation integral for the energy equation) derived from equilibrium pressure gradients for use in non-equilibrium pressure gradients.

---

<sup>1</sup>References are listed on page 43.

It is now proposed to return to the traditional  $\theta$ -H methods even for two-parameter velocity profiles in deriving shape parameter equations for the various integral methods. Analytical relations are obtained for the various shape parameters for two-parameter velocity profiles. Analytical expressions are also derived for the various shear-stress factors under equilibrium pressure gradients. Consideration is given to applying equilibrium shear-stress factors to non-equilibrium pressure gradients.

## VELOCITY SIMILARITY LAWS FOR BOUNDARY LAYER FLOW WITH PRESSURE GRADIENTS

The velocity similarity laws, the law of the wall and the velocity defect law, provide an analytical basis for pipe flow and boundary layer flow on flat plates (in zero pressure gradient). The extension to boundary-layer flow with pressure gradients may be made to proceed as follows:

### INNER LAW OR LAW OF THE WALL

In addition to pressure gradient  $dp/dx$ , the mean velocity component  $u$  parallel to the smooth wall (roughness and other effects may also be included) is considered to be dependent on the usual local quantities: normal distance from the wall  $y$ , shearing stress at the wall  $\tau_w$ , density  $\rho$  and kinematic viscosity  $\nu$  of the fluid or

$$u = f \left[ \tau_w, \rho, \nu, y, \frac{dp}{dx} \right] \quad (1)$$

Non-dimensional ratios may be formed

$$\frac{u}{u_\tau} = f \left[ \frac{u_\tau y}{\nu}, \frac{\nu}{\rho u_\tau^3} \frac{dp}{dx} \right] \quad (2)$$

where  $u_\tau = \sqrt{\tau_w/\rho}$ , shear velocity.

### OUTER LAW OR VELOCITY DEFECT LAW

In the boundary layer away from the wall, the velocity defect  $U - u$  at a relative position in the boundary  $y/\delta$  develops as a consequence of

the cumulative effect of the pressure gradients which may be represented by some appropriate characteristic value of the shearing stress  $\tau^*$ . Mickley et al.<sup>5</sup> uses the value of shearing stress  $\tau$  at the inner edge of the outer layer as  $\tau^*$ ; for adverse pressure gradients  $\tau^*$  becomes the maximum value of  $\tau$  and for zero pressure gradient  $\tau_w$ . Likewise McDonald and Stoddart<sup>6</sup> use the maximum value of  $\tau$  as  $\tau^*$  for adverse pressure gradients. The density  $\rho$  is an additional physical parameter. Analytically then

$$U - u = f[y/\delta, \rho, \tau^*] \quad (3)$$

where  $\delta$  is boundary layer thickness and  $U$  is value of  $u$  at  $y = \delta$ .

Non-dimensionally

$$\frac{U-u}{u_*} = F\left[\frac{y}{\delta}\right] \quad (4)$$

where  $u_*$  is  $\sqrt{\tau^*/\rho}$ .

For flat plates (zero pressure gradient)  $\tau^* \rightarrow \tau_w$  and  $u_{\tau^*} \rightarrow u_\tau$ .

#### LOGARITHMIC LAW

Within the boundary layer the ranges of validity of the inner and outer laws are considered to overlap which results in logarithmic functions for both the inner and outer laws for the common region. Equating the derivatives of  $u$  with respect to  $y$  for the inner and outer laws, Equations (2) and (4), produces

$$\left(\frac{u_\tau y}{\nu}\right) \frac{\partial(u/u_\tau)}{\partial(u_\tau y/\nu)} = -\frac{u_*}{u_\tau} \left(\frac{y}{\delta}\right) \frac{\partial\left(\frac{U-u}{u_*}\right)}{\partial(y/\delta)} = A \quad (5)$$

Then integrating

$$\frac{u}{u_\tau} = A \ln \frac{u_\tau y}{\nu} + B_1 \left[ \frac{\nu}{\rho u_\tau^3} \frac{dp}{dx} \right] \quad (6)$$

Experimentally Patel<sup>7</sup> shows  $B_1 = B_{1,0}$  for  $(\nu/\rho u_\tau^3) (dp/dx) < 0.01$  where  $B_{1,0} = B_1[0]$  for flat plates (zero pressure gradient).

Also

$$\frac{U-u}{u_{\tau}^*} = - \frac{u_{\tau}}{u_{\tau}^*} A \ln \frac{y}{\delta} + B_{2,0} \quad (7)$$

or

$$\frac{U-u}{u_{\tau}} = - A \ln \frac{y}{\delta} + \frac{u_{\tau}^*}{u_{\tau}} B_{2,0} \quad (8)$$

or

$$\frac{U-u}{u_{\tau}} = - A \ln \frac{y}{\delta} + B_2 \quad (9)$$

where  $B_2 = \left( \frac{u_{\tau}^*}{u_{\tau}} \right) B_{2,0}$  and  $B_{2,0} = B_2$  for flat plates (zero pressure gradient).

The value of  $B_2$  then depends on the history of the effects of the pressure gradients up to the station being considered or  $B_2 = f[x]$ . For specially adjusted pressure gradients termed equilibrium pressure gradients,  $B_2$  can be held constant with respect to  $x$ . Boundary layers on flat plates in zero pressure gradient may be considered as a special case of equilibrium boundary layers with  $B_2 = B_{2,0}$ . In general even for equilibrium pressure gradients  $B_2 \neq B_{2,0}$ . Towards separation  $B_2 \rightarrow \infty$ .

#### LAW OF THE WAKE

It was observed by Coles<sup>8</sup> that the experimental data for the outer law outside the overlapping region had similarity in its deviation from the logarithmic law such that

$$\frac{U-u}{u_{\tau}} = - A \ln \frac{y}{\delta} + B_2 \left( 1 - \frac{1}{2} w \left[ \frac{y}{\delta} \right] \right) \quad (10)$$

where  $w[y/\delta]$  is considered as a universal function termed the wake function. The wake function given in tabular form by Coles<sup>8</sup> was fitted with a sigmoidal function by Hinze<sup>9</sup>

$$w = 1 + \sin \left[ \frac{y}{\delta} \pi - \frac{\pi}{2} \right] = 1 - \cos \left[ \frac{y}{\delta} \pi \right] \quad (11)$$

A polynomial fit is given by Moses<sup>10</sup> as

$$w = 2 \left[ 3 \left( \frac{y}{\delta} \right)^2 - 2 \left( \frac{y}{\delta} \right)^3 \right] \quad (12)$$

In an earlier analysis Rotta<sup>11</sup> used as a first approximation a linear function

$$w = 2 \frac{y}{\delta} \quad (13)$$

The wake function is normalized so  $\int_0^1 w d[y/\delta] = 1$ . Also  $w[1] = 2$ .

#### BOUNDARY LAYER PARAMETERS

##### GENERAL

Analytical models of the velocity profile are designated one-parameter if

$$\frac{u}{U} = f \left[ \frac{y}{\theta}, H \right] \quad (14)$$

and two-parameter if

$$\frac{u}{U} = f \left[ \frac{y}{\theta}, H, R_{\theta} \right] \quad (15)$$

where  $\theta$ , momentum thickness =  $\int_0^{\delta} (1 - u/U) (u/U) dy$ ;

$\delta^*$ , displacement thickness =  $\int_0^{\delta} (1 - u/U) dy$ ;

$H$ , shape parameter (due to Gruschwitz) =  $\delta^*/\theta$ ; and

$R_{\theta}$ , momentum thickness Reynolds number =  $U\theta/\nu$ .

##### POWER LAW

An example of a one-parameter velocity profile is the familiar power law

$$\frac{u}{U} = \left( \frac{y}{\delta} \right)^n \quad (16)$$

which produces

$$H = 2n + 1 \quad (17)$$

$$\frac{\delta^*}{\delta} = \frac{H - 1}{H + 1} \quad (18)$$

$$\frac{\theta}{\delta} = \frac{H - 1}{H(H+1)} \quad (19)$$

Then

$$\frac{u}{U} = \left[ \left( \frac{y}{\theta} \right) \frac{(H - 1)}{H(H+1)} \right]^{\frac{H-1}{2}} \quad (20)$$

As shown in Figure 5 of Reference 12 the power law model provides a surprisingly close fit to experimental velocity profiles in pressure gradients.

#### VELOCITY SIMILARITY LAWS

The velocity similarity laws provide a two-parameter velocity profile.

A useful shape parameter is Rotta's shape parameter (also called Clauser's shape parameter)

$$G \equiv \frac{\int_0^1 \left( \frac{U-u}{u_\tau} \right)^2 d\left(\frac{y}{\delta}\right)}{\int_0^1 \left( \frac{U-u}{u_\tau} \right) d\left(\frac{y}{\delta}\right)} \quad (21)$$

If the velocity defect law is assumed to hold also to the wall, G is constant for constant  $B_2$  of equilibrium boundary layers. Also from the definitions

$$G = \frac{U}{u_\tau} \left( \frac{H-1}{H} \right) = \left( \frac{\tau_w}{\rho U^2} \right)^{-\frac{1}{2}} \left( \frac{H-1}{H} \right) \quad (22)$$

From the law of the wake

$$G = \frac{4 A^2 + 3.18 A B_2 + 0.75 B_2^2}{2 A + B_2} \quad (23)$$

#### WALL SHEARING STRESS

The coefficient of wall shearing stress or local skin friction is expressed by

$$\frac{\tau_w}{\rho U^2} = f[H, R_\theta] \quad (24)$$

As shown by Rotta<sup>1</sup> and Patel,<sup>13</sup> the law of the wake provides an implicit relationship for  $(\tau_w/\rho U^2)[H, R_\theta]$ . However an explicit relationship obtained in Reference 12 is a generalization of the procedure of Ludwig and Tillmann<sup>14</sup> or

$$\frac{\tau_w}{\rho U^2} = \left( \frac{\tau_w}{\rho U^2} \right)_o \left( \frac{\gamma}{\gamma_o} \right)^{\frac{4}{H_o+1}} \quad (25)$$

where  $(\tau_w/\rho U^2)_o$  is the flat plate value for the same  $R_\theta$ ,  $(\tau_w/\rho U^2)_o = f[R_\theta]$ .  $\gamma = (u/U)_{y=\theta}$ ,  $\gamma_o$ , and  $H_o$  are the flat plate values.

Empirically both Uram<sup>15</sup> and Felsch<sup>16</sup> express  $\gamma$  as

$$\gamma = c_1 - c_2 \log H \quad (26)$$

(Here log is taken as  $\log_{10}$ .)

Uram gives  $c_1 = 0.9058$  and  $c_2 = 1.818$  while Felsch gives  $c_1 = 0.93$  and  $c_2 = 1.95$ .

At separation,  $\tau_w/\rho U^2 = 0$ , Uram's constants give  $H = 3.15$  and Felsch's constants give  $H = 3$  which is closer to test data than  $H = 4$  given by the law of the wake. Nash<sup>17</sup> uses  $H = 3$  at separation in a modified skin-friction formula for pressure gradients. Also Equation (25) with Felsch's constants for  $\gamma$  gives a very close fit to Nash's recommendation.

## WALL SHEARING STRESS FOR FLAT PLATES

As shown in Reference 12,  $\left(\frac{\tau_w}{\rho U^2}\right)_o [R_\theta]$  is derived from the Schoenherr formula for the total drag of flat plates as

$$\left(\frac{\tau_w}{\rho U^2}\right)_o = \frac{0.0146}{\left(\log [2 R_\theta]\right) \left(1/2 \log [2 R_\theta] + 0.4343\right)} \quad (27)$$

Formulas of type

$$\left(\frac{\tau_w}{\rho U^2}\right)_o = \frac{1}{\left(\alpha_1 R_\theta + \alpha_2\right)^2} \quad (28)$$

may also be derived as follows:

Adding the overlapping inner and outer logarithmic laws, Equations (6) and (9) produces

$$\frac{U}{u_\tau} = A \ln \frac{u_\tau \delta}{\nu} + B_1 + B_2 \quad (29)$$

Since  $u_\tau \delta / \nu = (u_\tau / U) (\delta / \theta) R_\theta$  by definition

$$\frac{U}{u_\tau} = -A \ln \frac{U}{u_\tau} + A \ln R_\theta - A \ln \frac{\theta}{\delta} + B_1 + B_2 \quad (30)$$

If the velocity defect law is assumed to hold to the wall

$$\frac{\theta}{\delta} = \frac{u_\tau}{U} I_1 \left(1 - \frac{u_\tau}{U} G\right) = \frac{u_\tau}{U} \frac{I_1}{H} \quad (31)$$

where

$$I_1 \equiv \int_0^1 \left(\frac{U-u}{u_\tau}\right) d\left(\frac{y}{\delta}\right) \quad (32)$$

From the law of the wake



$$I_1 = A + \frac{1}{2} B_2 \quad (33)$$

Then for flat plates denoted by subscript o

$$\frac{U}{u_\tau} = A \ln R_\theta + A \ln H_o - A \ln I_{1,0} + B_1 + B_{2,0} \quad (34)$$

If

$$\ln H_o \cong c_3 + c_4 \frac{U}{u_\tau} \quad (35)$$

then

$$\frac{U}{u_\tau} = \alpha_1 \log R_\theta + \alpha_2 \quad (36)$$

or

$$\left( \frac{\tau_w}{\rho U^2} \right)_o = \frac{1}{(\alpha_1 \log R_\theta + \alpha_2)^2} \quad (37)$$

where

$$\alpha_1 = \frac{2.3026 A}{1 - c_4 A} \quad (38)$$

and

$$\alpha_2 = \frac{B_1 + B_{2,0} + c_3 A - \ln(A + 1/2 B_{2,0})}{1 - c_4 A}$$

#### SHAPE PARAMETER FOR FLAT PLATES

Since from Equation (22)

$$\frac{H_o}{H_o - 1} = \frac{1}{G_o \sqrt{\left( \frac{\tau_w}{\rho U^2} \right)_o}} \quad (40)$$

then from Equation (37)

$$\frac{H_o}{H_o - 1} = \frac{\alpha_1}{G_o} \log R_\theta + \frac{\alpha_2}{G_o} \quad (41)$$

or

$$\frac{H_o}{H_o - 1} = \mathcal{A} \log R_\theta + \mathcal{B} \quad (42)$$

or

$$H_o = \left[ 1 - (\mathcal{A} \log R_\theta + \mathcal{B})^{-1} \right]^{-1} \quad (43)$$

where  $\mathcal{A} = \frac{\alpha_1}{G_o}$  and  $\mathcal{B} = \frac{\alpha_2}{G_o}$

#### ENERGY THICKNESS AND SHAPE PARAMETER

The energy thickness  $\theta^*$  is defined as

$$\theta^* \equiv \int_0^\delta \frac{u}{U} \left[ 1 - \left( \frac{u}{U} \right)^2 \right] dy \quad (44)$$

The energy shape parameter  $H^*$  is defined as

$$H^* \equiv \theta^* / \theta \quad (45)$$

For one-parameter velocity profiles

$$H^* = f [H] \quad (46)$$

while for two-parameter velocity profiles

$$H^* = f [H, R_\theta] \quad (47)$$

The simplest relation is from the power-law velocity profile, a one-parameter velocity profile, which is

$$H^* = \frac{4H}{3H-1} = \frac{\frac{4}{3} H}{H - \frac{1}{3}} \quad (48)$$

Closer empirical fits (one-parameter) have been obtained by various investigators.<sup>18-22</sup>

From Weighardt:

$$H^* = \frac{1.269 H}{H - 0.379} \quad 1.25 < H < 1.7 \quad (49)$$

From Fernholz:

$$H^* = \frac{1.272 H}{H - 0.37} + 5.4 \left(\frac{H}{10}\right)^4 \quad \begin{array}{l} 1.35 < H < 2.8 \\ 5 \times 10^3 < R_\theta < 2.5 \times 10^4 \end{array} \quad (50)$$

From Moses et al.:

$$H^* = \frac{1.02 + 0.87 H + 0.095 H^2}{H} \quad (51)$$

From Goldberg:

$$H^* = \frac{3.6 H}{2.78 H - 1} \quad (52)$$

From Nicoll and Escudier:

$$H^* = 1.431 - \frac{0.0971}{H} + \frac{0.775}{H^2} \quad \begin{array}{l} 1.25 < H < 2.8 \\ 10^3 < R_\theta < 8 \times 10^4 \end{array} \quad (53)$$

A two-parameter relation  $H^* = f[H, R_\theta]$  may be obtained from the velocity similarity laws as follows. A velocity-defect law energy shape parameter  $G^*$  is defined as follows

$$G^* \equiv \frac{\int_0^1 \left(\frac{U-u}{u_\tau}\right)^3 d\left(\frac{y}{\delta}\right)}{\int_0^1 \left(\frac{U-u}{u_\tau}\right) d\left(\frac{y}{\delta}\right)} \quad (54)$$

From the appropriate definitions

$$H^* = \frac{\left(\frac{u_\tau}{U}\right)^2 G^* - 3\left(\frac{u_\tau}{U}\right) G + 2}{1 - \left(\frac{u_\tau}{U}\right) G} \quad (55)$$

Then, from Equation (22)

$$H^* = 3 - H + \left(\frac{G^*}{G^2}\right) \frac{(H-1)^2}{H} \quad (56)$$

Equation (56) was also obtained by Rotta.<sup>11</sup>

From the law of the wake

$$G^* = \frac{12 A^3 + 11.04 A^2 B_2 + 4.71 A B_2^2 + 0.63 B_2^3}{2 A + B_2} \quad (57)$$

$G^*$  and  $G$  are then related through Equations (57) and (23) with  $B_2$  being the implicit parameter. At separation  $B_2 \rightarrow \infty$  and  $G^*/G^2 \rightarrow 1.12$ . An explicit numerical fit gives closely

$$\frac{G^*}{G^2} = 1.12 + \frac{1.5}{G} \quad (58)$$

Then with Equation (56) and Equation (22)

$$H^* = 3 - H + 1.12 \frac{(H-1)^2}{H} + 1.5 (H-1) \sqrt{\frac{\tau_w}{\rho U^2}} \quad (59)$$

where the variation with  $R_\theta$  is obtained through  $\tau_w/\rho U^2$  given by Equation (25).

The comparison in Figure 1 shows excellent agreement between the two-parameter relation and the empirical one-parameter relations of Fernholz and of Nicoll and Escudier.

#### ENTRAINMENT THICKNESS AND SHAPE PARAMETER

The entrainment thickness is defined as  $\delta - \delta^*$  and the entrainment shape parameter  $\tilde{H}$  as

$$\tilde{H} \equiv \frac{\delta - \delta^*}{\theta} = \frac{\delta}{\theta} - H \quad (60)$$

For one-parameter velocity profiles

$$\tilde{H} = f [H] \quad (61)$$

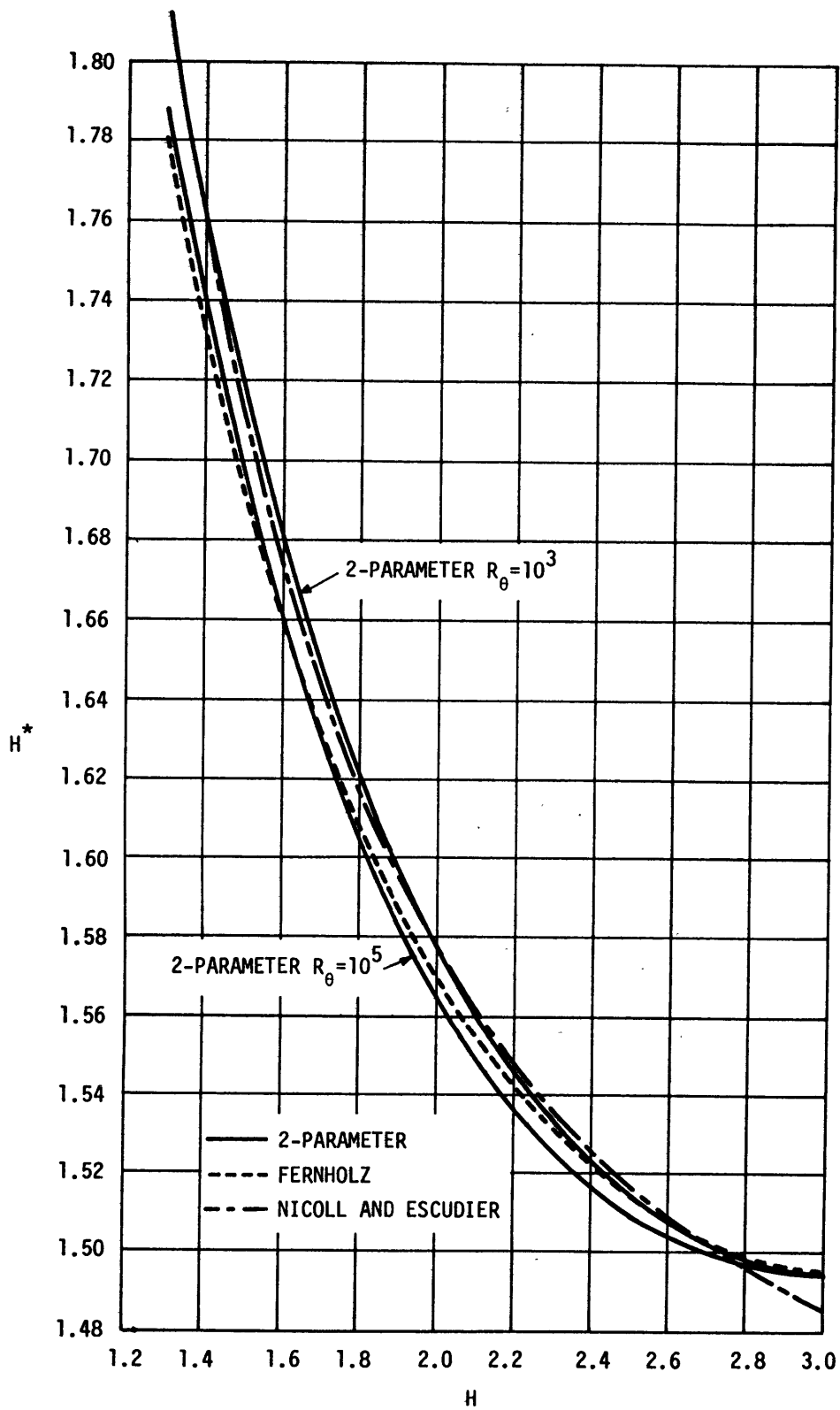


Figure 1 - Comparison of Energy Shape Parameters

while for two-parameter velocity profiles

$$\tilde{H} = f [H, R_\theta] \quad (62)$$

The power law velocity profile gives

$$\tilde{H} = \frac{2H}{H-1} \quad (63)$$

An empirical fit by Head<sup>4</sup> yields the one-parameter relation

$$\tilde{H} = 1.535 (H-0.7)^{-2.715} + 3.3 \quad (64)$$

A relation  $\tilde{H} = f [H, R_\theta]$  may be obtained from the velocity similarity laws. From the definition

$$\frac{\theta}{\delta} = \frac{u_\tau}{U} I_1 \left( 1 - \frac{u_\tau}{U} G \right) \quad (65)$$

Then from Equation (22)

$$\frac{\theta}{\delta} = \frac{I_1}{G} \left( \frac{H-1}{H^2} \right) \quad (66)$$

and from Equation (60)

$$\tilde{H} = \left( \frac{G}{I_1} \right) \left( \frac{H^2}{H-1} \right) - H \quad (67)$$

This relation was also obtained by Michel et al.<sup>4</sup>

$I_1$  and  $G$  may be related through Equations (33) and (23) with  $B_2$  being the implicit parameter. At separation  $B_2 \rightarrow \infty$  and  $G/I_1 \rightarrow 1.5$ . A close numerical fit gives

$$\frac{G}{I_1} = 1.5 + \frac{3.8}{G^{3/2}} \quad (68)$$

Then from Equation (67)

$$\tilde{H} = \frac{H(H+2)}{2(H-1)} + \frac{3.8 H^{7/2}}{(H-1)^{5/2}} \left( \frac{\tau_w}{\rho U^2} \right)^{3/4} \quad (69)$$

For  $H = 3$ ,  $\tau_w/\rho U^2 = 0$  and  $\tilde{H} = 3.75$ .

The comparison in Figure 2 shows close agreement between the two-parameter values and the empirical fit of Head.

### EQUILIBRIUM PRESSURE GRADIENTS

#### EFFECT OF PRESSURE GRADIENT PARAMETER

It has been shown theoretically by Rotta and experimentally by Clauser that similarity is maintained if the pressure gradient parameter

$$\beta \equiv \frac{\delta^*}{\tau_w} \frac{dp}{dx} \quad (70)$$

is kept constant with respect to  $x$  or  $d\beta/dx = 0$ .

Then  $G$  and consequently  $B_2$  are constant. Empirically Nash<sup>23</sup> obtains

$$G = 6.1 \sqrt{1.81 + \beta} - 1.7 \quad (71)$$

Felsch<sup>16</sup> obtains

$$G = 6 \sqrt{1.8 + \beta} - 1.5 \quad (72)$$

and Alber<sup>4,24</sup>

$$\begin{aligned} G &= 6.1 \sqrt{1.81 + \beta} - 0.40, & \beta &\geq 0 \\ G &= 6.5 \beta + 7.8067, & \beta &< 0 \end{aligned} \quad (73)$$

#### SHAPE PARAMETER

For equilibrium pressure gradients Equation (34) is stated as

$$\frac{U}{u_\tau} = A \ln R_\theta + A \ln H - A \ln I_1 + B_1 + B_2 \quad (74)$$

From Equation (22)

$$\frac{H}{H-1} = \frac{1}{G} \frac{U}{u_\tau} = \frac{A}{G} \ln R_\theta + \frac{A}{G} \ln H - \frac{A}{G} \ln I_1 + \frac{B_1}{G} + \frac{B_2}{G} \quad (75)$$

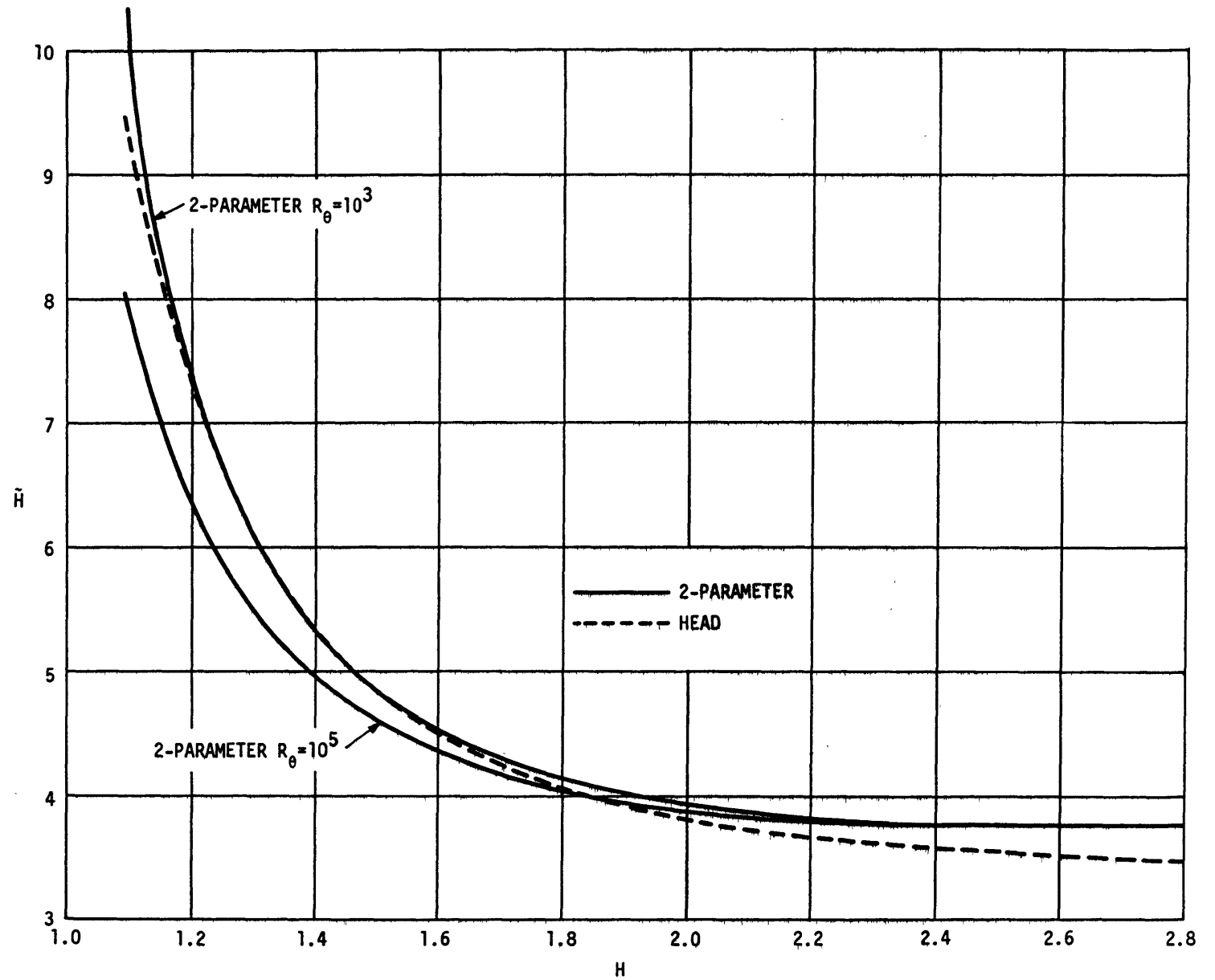


Figure 2 - Comparison of Entrainment Shape Parameters



In general A is constant and  $B_1$  is constant for smooth surfaces. For equilibrium boundary layers G,  $I_1$  and  $B_2$  are constant with respect to x for a particular  $\beta$ .

Differentiating Equation (75) with respect to x produces

$$\theta \frac{dH}{dx} = - \frac{A(H-1)^2 \left[ H + (H+1)\beta \right] \frac{\tau_w}{\rho U^2}}{G H + A(H-1)^2} \quad (76)$$

for equilibrium boundary layers.

An alternate form from Equation (22) is

$$\theta \frac{dH}{dx} = - \frac{G H \frac{\partial}{\partial \ln R_\theta} \left( \frac{\tau_w}{\rho U^2} \right)^{\frac{1}{2}} \left[ H + (H+1)\beta \right] \frac{\tau_w}{\rho U^2}}{G H^2 \frac{\partial}{\partial H} \left( \frac{\tau_w}{\rho U^2} \right)^{\frac{1}{2}} - 1} \quad (77)$$

where  $\tau_w/\rho U^2$  is given by Equation (25).

## INTEGRAL METHODS

### GENERAL

Integral methods for solving the turbulent boundary layers in pressure gradients refer to methods based on integrated forms of the equation of motion (momentum equation) and/or the equation of continuity, using various weighting factors which for incompressible two-dimensional flow are

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = U \frac{dU}{dx} + \frac{1}{\rho} \frac{\partial \tau}{\partial y} \quad (78)$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad (79)$$

The shearing stress  $\tau$  includes both laminar and turbulent contributions

$$\tau = \mu \frac{du}{dy} - \rho \overline{u'v'} \quad (80)$$

where  $\overline{u'v'}$  = Reynolds turbulent shear stress. The Reynolds turbulent normal stresses are not included though these may become quite significant close to separation. Also close to separation  $\partial p/\partial y \neq 0$ .

The classical integral form is von Kármán's momentum equation obtained by integrating the equation of motion without using any weighting factor

$$\frac{d\theta}{dx} + (H+2) \frac{\theta}{U} \frac{dU}{dx} = \frac{\tau_w}{\rho U^2} \quad (81)$$

The purpose of other integrated forms is to obtain eventually the variation of H with x or dH/dx. The energy equation uses u as a weighting factor. The entrainment equation integrates the equation of continuity. The moment-of-momentum equation uses y as the weighting factor. The partial momentum equations partially integrate the equation of motion to differently specified sublevels within the boundary layer. Details of these equations follow.

The shape parameter equation may take the following forms

$$\theta \frac{dH}{dx} = - M[H, R_\theta] \frac{\theta}{U} \frac{dU}{dx} + N[H, R_\theta] \frac{\tau_w}{\rho U^2} - P[H, R_\theta] C_S \quad (82)$$

or

$$\theta \frac{dH}{dx} = - M[H, R_\theta] \frac{\theta}{U} \frac{dU}{dx} - \left( P[H, R_\theta] \hat{C}_S - N[H, R_\theta] \right) \frac{\tau_w}{\rho U^2} \quad (83)$$

where M, N, P are coefficients,  $C_S$  is the generalized shear-stress factor and

$$\hat{C}_S = \frac{C_S}{\tau_w/\rho U^2} \quad (84)$$

## EQUILIBRIUM SHEAR-STRESS FACTORS

To obtain  $C_S[H, R_\theta]$  for equilibrium boundary layers,  $\theta dH/dx$  from Equation (83) is first converted to form

$$\theta \frac{dH}{dx} = \left( \frac{\beta M}{H} - P \hat{C}_S + N \right) \frac{\tau_w}{\rho U^2} \quad (85)$$

and equated to Equation (76) to produce

$$\hat{C}_{S,e} = \frac{\beta M}{HP} + \frac{N}{P} + \frac{A(H-1)^2 [H+(H+1)\beta]}{[GH + A(H-1)^2] P} \quad (86)$$

where  $\hat{C}_{S,e}$  represents the equilibrium value of  $\hat{C}_S$ .

$\hat{C}_{S,e}$  may be considered a function of  $H$  and  $R_\theta$  if  $\beta$  is related to  $G$  by Equation (71) and  $G$  is a function of  $H$  and  $R_\theta$  through Equation (22).

## ENERGY METHOD

### GENERAL

The energy equation is obtained by multiplying the equation of motion Equation (78) by  $u$  and integrating over the whole boundary layer

$$\frac{d}{dx} \left( U^3 \theta^* \right) = \frac{2}{\rho} \int_0^\delta \tau \frac{\partial u}{\partial y} dy \quad (87)$$

In terms of energy shape parameter  $H^* \equiv \theta^*/\theta$  and employing the momentum equation for  $d\theta/dx$

$$\theta \frac{dH^*}{dx} = (H-1)H^* \frac{\theta}{U} \frac{dU}{dx} - H^* \frac{\tau_w}{\rho U^2} + C_D \quad (88)$$

where

$$C_D \equiv \frac{2}{\rho U^3} \int_0^\delta \tau \frac{\partial u}{\partial y} dy \quad (89)$$

called the dissipation integral, the shear-stress work integral or the production integral.

## SHAPE PARAMETER EQUATION

The objective is to convert the energy shape parameter equation, Equation (88), to a shape parameter equation of forms Equations (82) or (83).

For a two-parameter velocity profile

$$H^* = H^* \left[ H, \ln R_\theta \right] \quad (90)$$

Then expanding into partial derivatives

$$\theta \frac{dH^*}{dx} = \frac{\partial H^*}{\partial H} \theta \frac{dH}{dx} + \frac{\partial H^*}{\partial \ln R_\theta} \theta \frac{d \ln R_\theta}{dx} \quad (91)$$

or

$$\theta \frac{dH^*}{dx} = \frac{\partial H^*}{\partial H} \theta \frac{dH}{dx} + \frac{\partial H^*}{\partial \ln R_\theta} \left[ \frac{\tau_w}{\rho U^2} - (H+1) \frac{\theta}{U} \frac{dU}{dx} \right] \quad (92)$$

and finally

$$\begin{aligned} \theta \frac{dH}{dx} = & \left[ (H-1)H^* + (H+1) \frac{\partial H^*}{\partial \ln R_\theta} \right] \frac{\partial H}{\partial H^*} \frac{\theta}{U} \frac{dU}{dx} \\ & - \left[ H^* + \frac{\partial H^*}{\partial \ln R_\theta} - \hat{C}_D \right] \frac{\partial H}{\partial H^*} \frac{\tau_w}{\rho U^2} \end{aligned} \quad (93)$$

where

$$\hat{C}_D \equiv 2 \int_0^1 \frac{\tau}{\tau_w} \frac{\partial (u/U)}{\partial (y/\delta)} d\left(\frac{y}{\delta}\right) \quad (94)$$

Then from Equation (82)

$$M = - \left[ (H-1)H^* + (H+1) \frac{\partial H^*}{\partial \ln R_\theta} \right] \frac{\partial H}{\partial H^*} \quad (95)$$

$$N = - \left( H^* + \frac{\partial H^*}{\partial \ln R_\theta} \right) \frac{\partial H}{\partial H^*} \quad (96)$$

and

$$P = - \frac{\partial H}{\partial H^*} \quad (97)$$

$C_{D,e}$  is obtained from Equation (86) since  $C_{D,e} = C_{S,e}$  for the energy equation.

#### POWER-LAW ENERGY THICKNESS

Here

$$H^* = f[H] = \frac{4H}{3H-1} \quad (48)$$

Then

$$\frac{dH^*}{dH} = - \frac{4}{(3H-1)^2} \quad (98)$$

$$M = H(H-1) (3H-1) \quad (99)$$

$$N = H(3H-1) \quad (100)$$

$$P = \frac{1}{4} (3H-1)^2 \quad (101)$$

and

$$\hat{C}_{D,e} = \frac{4[(H-1)\beta+H]}{3H-1} + \frac{4 A(H-1)^2 [H+(H+1)\beta]}{(3H-1)^2 [GH + A(H-1)^2]} \quad (102)$$

#### FERNHOLZ ENERGY THICKNESS

Here

$$H^* = \frac{1.272 H}{H - 0.37} + 5.4 \times 10^{-4} H^4 \quad (50)$$

Then

$$\frac{dH^*}{dH} = - \frac{0.4706}{(H-0.37)^2} + 2.16 \times 10^{-3} H^3 \quad (103)$$

$$M = \frac{(H-1)(H-0.37) \left[ 1.272 H + 5.4 \times 10^{-4} (H-0.37)H^4 \right]}{0.4706 - 2.16 \times 10^{-3} (H-0.37)^2 H^3} \quad (104)$$

$$N = \frac{M}{H-1} \quad (105)$$

$$P = - \frac{dH}{dH^*} = \frac{(H - 0.37)^2}{0.4706 - 2.16 \times 10^{-3} (H - 0.37)^2 H^3} \quad (106)$$

and

$$\hat{C}_{D,e} = \left[ \frac{(H-1)}{H} \beta + 1 \right] H^* - \left\{ \frac{A(H-1)^2 [H + (H+1)\beta]}{GH + A(H-1)^2} \right\} \frac{dH^*}{dH} \quad (107)$$

#### TWO-PARAMETER ENERGY THICKNESS

Here

$$H^* = 3 - H + \frac{1.12(H-1)^2}{H} + 1.5 (H-1) \sqrt{\frac{\tau_w}{\rho U^2}} \quad (59)$$

With Equation (25)

$$\frac{\partial H^*}{\partial H} = -1 + \frac{1.12(H^2-1)}{H^2} + 1.5 \sqrt{\frac{\tau_w}{\rho U^2}} \left[ 1 - \frac{2 c_2 (H-1)}{2.3(H_0+1) H \gamma} \right] \quad (108)$$

and with Equations (37) and (42)

$$\frac{\partial H^*}{\partial \ln R_\theta} = -1.5(H-1) \sqrt{\frac{\tau_w}{\rho U^2}} \left[ \frac{\alpha_1}{2.3} \left( \frac{\tau_w}{\rho U^2} \right)^{\frac{1}{2}} + \frac{2 \alpha (H_0-1)^2}{2.3(H_0+1)} \left( \frac{c_2}{2.3 H_0 \gamma_0} - \frac{\ln \frac{\gamma}{\gamma_0}}{H_0+1} \right) \right] \quad (109)$$

M, N, P, and  $C_{D,e}$  are obtained from Equations (95), (96), (97), and (86).

Figure 3 shows how close the M values are for the various formulations.

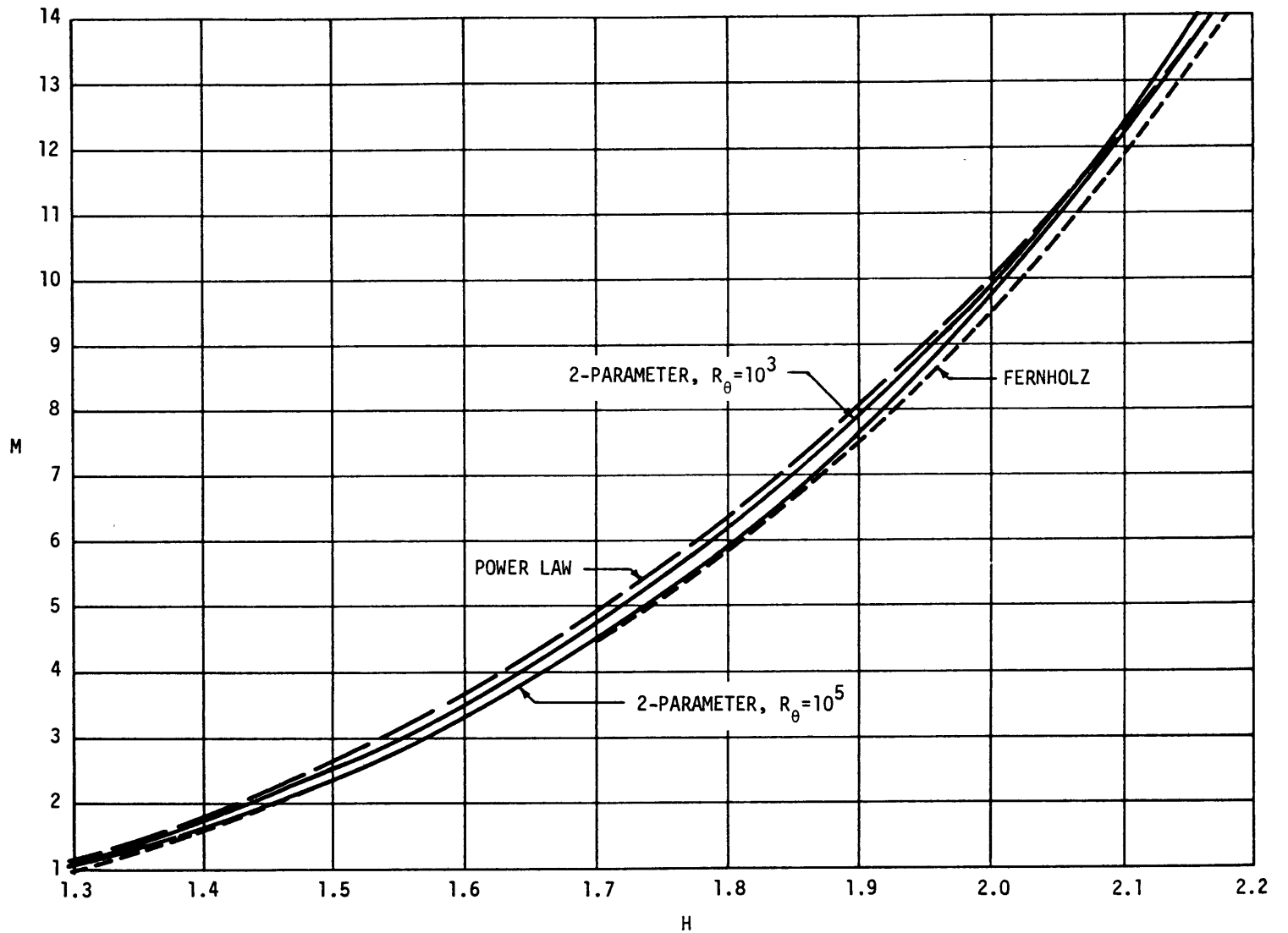


Figure 3 - Energy Method,  $M = f[H, R_\theta]$ ; Comparison of Various Procedures

EXISTING RELATIONS FOR DISSIPATION  
INTEGRAL

Truckenbrodt<sup>25</sup> approximates

$$C_D = \frac{0.0112}{R_\theta^{1/6}} \quad (110)$$

from the analyses of Rotta.<sup>11</sup>

Walz<sup>26</sup> suggests for equilibrium boundary layers

$$C_{D,e} = \frac{0.00962 + 0.1644 (H^* - 1.5)^{4.81}}{R_\theta (0.2317H - 0.2644)} \quad (111)$$

where  $H^*$  is determined by Fernholz, Equation (50).

A fit of experimental shear-stress data by Escudier and Spalding<sup>27</sup> results in

$$C_D = 1.094 \frac{\tau_w}{\rho U^2} + 0.004214 H - 0.004572 \quad (112)$$

Escudier et al.<sup>28</sup> propose

$$C_D = \frac{2}{3} (2 \zeta + 1) \frac{\tau_w}{\rho U^2} + 0.0113 (1 - \zeta)^{2.715} \quad (113)$$

where

$$\zeta = \frac{2}{3} H^* - 1 + \sqrt{\frac{2}{3} H^* \left( \frac{2}{3} H^* - 1 \right)} \quad (114)$$

Figure 4 compares the dissipation integral for the various formulations.

ENTRAINMENT METHOD

GENERAL

The entrainment equation was first proposed by Head<sup>29,4</sup> on the basis of physical reasoning regarding the growth or entrainment of the developing boundary layer. It has since been found out that it can be derived by integrating the equation of continuity, Equation (79). Michel et al.<sup>4</sup>



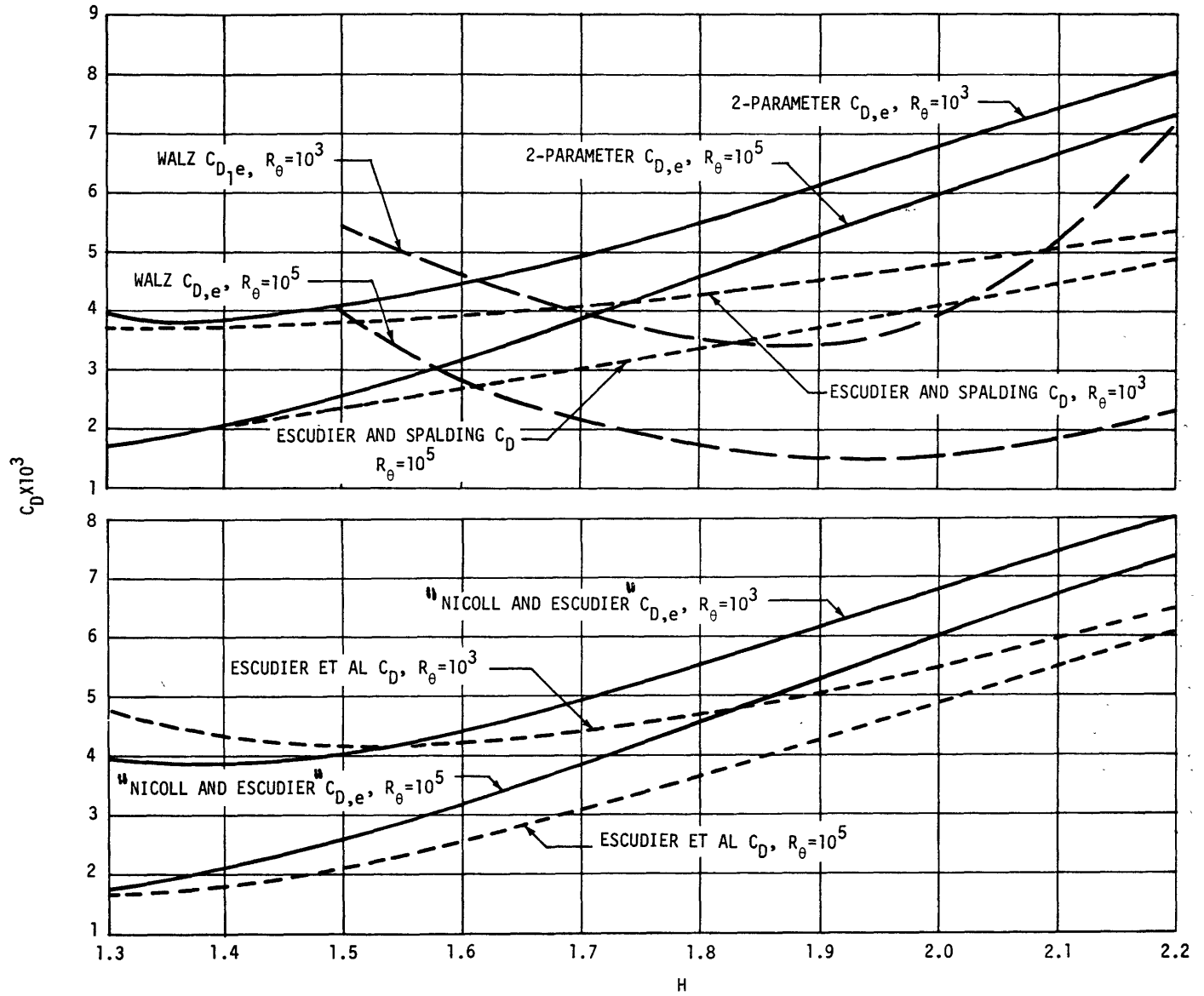


Figure 4 - Energy Method,  $C_D = f[H, R_\theta]$ ; Comparison of Various Procedures

introduced a shear-stress factor by considering the equation of motion Equation (78) at  $y = \delta$ . The entrainment equation then resembles the energy equation and is given by

$$\frac{1}{U} \frac{d}{dx} [U(\delta - \delta^*)] = - \frac{1}{\rho U} \left( \frac{\partial \tau}{\partial u} \right)_{\delta} = E \quad (115)$$

In terms of the entrainment shape parameter  $H = (\delta - \delta^*)/\theta$  and employing the momentum equation for  $d\theta/dx$

$$\theta \frac{d\tilde{H}}{dx} = (\tilde{H}+1)H \frac{\theta}{U} \frac{dU}{dx} - \tilde{H} \frac{\tau_w}{\rho U^2} + E \quad (116)$$

#### SHAPE PARAMETER EQUATION

The objective is to convert the entrainment shape parameter equation, Equation (116), to a shape parameter equation of forms Equations (82) or (83).

For a two-parameter velocity profile

$$\tilde{H} = f \left[ H, \ln R_{\theta} \right] \quad (117)$$

Then expanding into partial derivatives

$$\theta \frac{d\tilde{H}}{dx} = \frac{\partial \tilde{H}}{\partial H} \theta \frac{dH}{dx} + \frac{\partial \tilde{H}}{\partial \ln R_{\theta}} \theta \frac{d \ln R_{\theta}}{dx} \quad (118)$$

$$\theta \frac{d\tilde{H}}{dx} = \frac{\partial \tilde{H}}{\partial H} \theta \frac{dH}{dx} + \frac{\partial \tilde{H}}{\partial \ln R_{\theta}} \left[ \frac{\tau_w}{\rho U^2} - (H+1) \frac{\theta}{U} \frac{dU}{dx} \right] \quad (119)$$

and finally

$$\begin{aligned} \theta \frac{dH}{dx} &= (H+1) \left( \tilde{H} + \frac{\partial \tilde{H}}{\partial \ln R_{\theta}} \right) \frac{\partial H}{\partial \tilde{H}} \frac{\theta}{U} \frac{dU}{dx} \\ &\quad - \left( \tilde{H} + \frac{\partial \tilde{H}}{\partial \ln R_{\theta}} - \hat{E} \right) \frac{\partial H}{\partial \tilde{H}} \frac{\tau_w}{\rho U^2} \end{aligned} \quad (120)$$

where

$$\hat{E} \equiv \frac{E}{\frac{\tau_w}{\rho U^2}} = - \left[ \frac{\partial(\tau/\tau_w)}{\partial(u/U)} \right]_{\delta} \quad (121)$$

Then from Equation (82)

$$M = - (H+1) \left( \tilde{H} + \frac{\partial \tilde{H}}{\partial \ln R_{\theta}} \right) \frac{\partial H}{\partial \tilde{H}} \quad (122)$$

$$N = - \left( \tilde{H} + \frac{\partial \tilde{H}}{\partial \ln R_{\theta}} \right) \frac{\partial H}{\partial \tilde{H}} \quad (123)$$

$$P = - \frac{\partial H}{\partial \tilde{H}} \quad (124)$$

and  $\hat{E}_e$  is obtained from Equation (86) since  $\hat{E}_e = \hat{C}_{S,e}$  for the entrainment method.

The actual evaluation of  $\hat{E}_e$  and the accompanying shape parameter equation depends on the particular relation for entrainment thickness. Some examples are now presented:

#### POWER-LAW ENTRAINMENT THICKNESS

Here

$$\tilde{H} = \frac{2H}{H-1} \quad (63)$$

Then

$$\frac{d\tilde{H}}{dH} = - \frac{2}{(H-1)^2} \quad (125)$$

$$M = - H(H^2-1) \quad (126)$$

$$N = H(H-1) \quad (127)$$

$$P = \frac{(H-1)^2}{2} \quad (128)$$

and

$$\hat{E}_e = \frac{2H}{(H-1)} \left[ \frac{(H+1)\beta}{H} + 1 \right] + \frac{2A [H+(H+1)\beta]}{GH + A(H-1)^2} \quad (129)$$

## HEAD ENTRAINMENT THICKNESS

Here

$$\tilde{H} = 1.535 (H-0.7)^{-2.715} + 3.3 \quad (64)$$

Then

$$\frac{d\tilde{H}}{dH} = -4.168 (H-0.7)^{-3.715} \quad (130)$$

$$M = 0.3683 (H+1)(H-0.7) \left[ 1 + 2.15 (H-0.7)^{2.715} \right] \quad (131)$$

$$N = 0.3683 (H-0.7) \left[ 1 + 2.164 (H-0.7)^{2.715} \right] \quad (132)$$

$$P = \frac{(H-0.7)^{3.715}}{4.168} \quad (133)$$

and

$$E_e = \left[ \frac{(H+1)\beta}{H} + 1 \right] \tilde{H} - \left\{ \frac{A(H-1)^2 H + (H+1)\beta}{GH + A(H-1)^2} \right\} \quad (134)$$

## TWO-PARAMETER ENTRAINMENT THICKNESS

Here

$$\tilde{H} = \frac{H(H+2)}{2(H-1)} + \frac{3.8 H^{7/2}}{(H-1)^{5/2}} \left( \frac{\tau_w}{\rho U^2} \right)^{3/4} \quad (69)$$

With Equation (25)

$$\frac{\partial \tilde{H}}{\partial H} = \frac{H^2 - 2H - 2}{2(H-1)^2} + \frac{3.8 H^{7/2}}{(H-1)^{5/2}} \left( \frac{\tau_w}{\rho U^2} \right)^{3/4} \left[ \frac{2H-7}{2H(H-1)} - \frac{3c_2}{2.3(H_0+1)HY} \right] \quad (135)$$

and with Equations (37) and (42)

$$\frac{\partial \tilde{H}}{\partial \ln R_\theta} = - \frac{5.7 H^{7/2}}{(H-1)^{5/2}} \left( \frac{\tau_w}{\rho U^2} \right)^{3/4} \left[ \frac{\alpha_1}{2.3} \left( \frac{\tau_w}{\rho U^2} \right)^{1/2} + \frac{2\alpha(H_0-1)^2}{2.3(H_0+1)} \left( \frac{c_2}{2.3H_0\gamma_0} - \frac{\ln \frac{\gamma}{\gamma_0}}{H_0+1} \right) \right] \quad (136)$$

M, N, P, and  $E_e$  are obtained from Equations (122), (123), (124), and (86).

Figure 5 shows large differences in M between the various formulations.

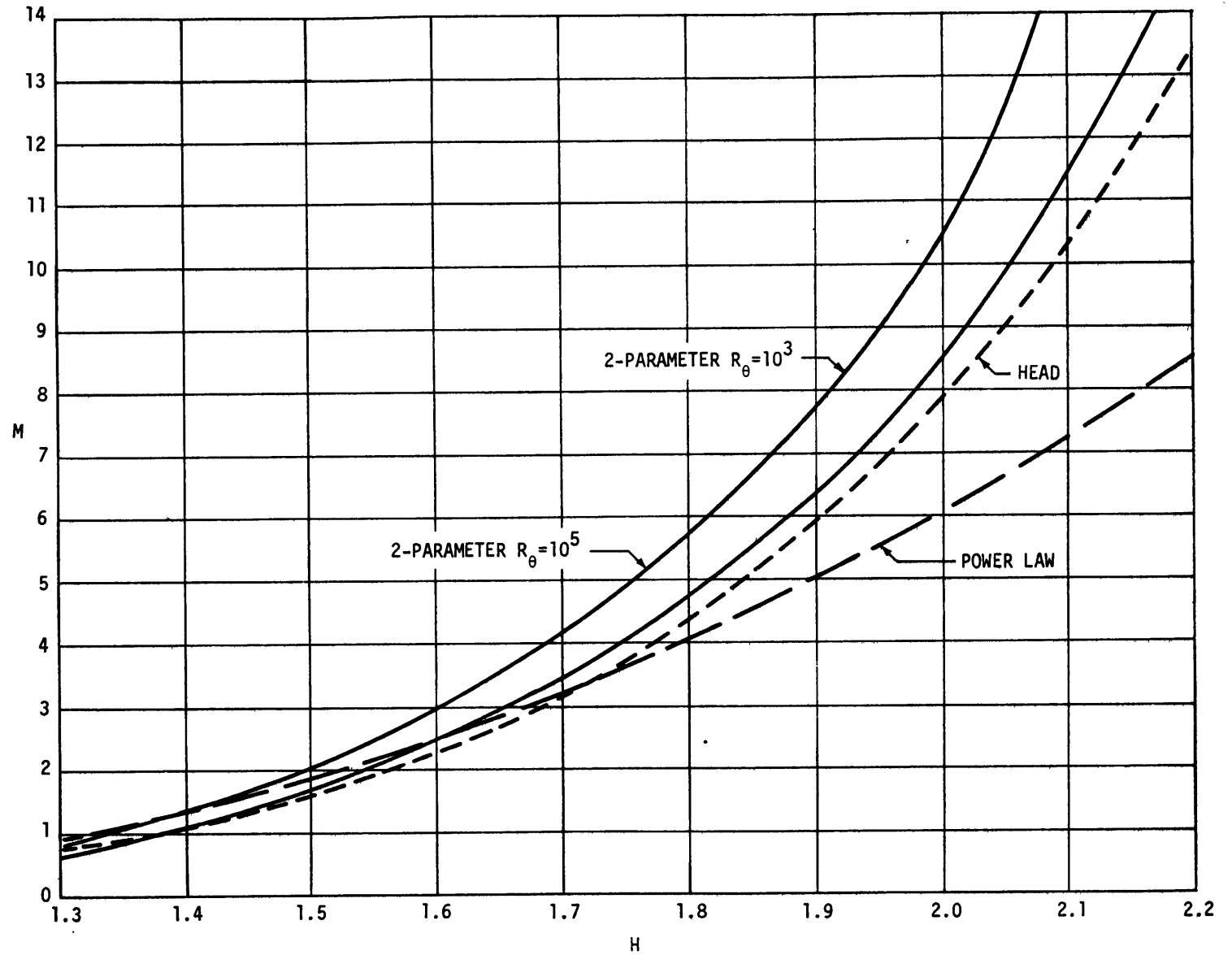


Figure 5 - Entrainment Method,  $M = f[H, R_\theta]$ ; Comparison of Various Procedures

## EXISTING RELATIONS FOR ENTRAINMENT FACTOR

Empirically Head<sup>4</sup> obtains

$$E = 0.0306 \left[ 1.535(H-0.7)^{-2.715} + 0.3 \right]^{-0.653} \quad (137)$$

Also Nicoll and Ramaprian<sup>30</sup> include Statford's data for separating flow and obtain an empirical fit

$$E = 0.035 \sqrt{H-1.25} \quad (138)$$

Figure 6 compares the various entrainment factors.

## PARTIAL MOMENTUM METHODS

### GENERAL

Another group of integrated equations which are transformable into shape parameter equations may be obtained by integrating the equation of motion, Equation (78), to some intermediate value of  $y$ , say  $s[x]$ .  $s$  may be  $s = m\delta$  where  $m$  is a constant which was used by Moses<sup>4</sup> or  $s = \theta$  which was used by Furuya and Nakamura.<sup>4</sup> Another possibility is  $s[u/U = \text{const}]$  which will not be treated here.

Integrating the equation of motion, Equation (78), to  $y = s$  produces

$$\begin{aligned} \frac{d}{dx} \int_0^s \left(\frac{u}{U}\right)^2 dy - \frac{u_s}{U} \int_0^s \frac{u}{U} dy + \left[ 2 \int_0^s \left(\frac{u}{U}\right)^2 dy - \frac{u_s}{U} \int_0^s \frac{u}{U} dy - s \right] \frac{1}{U} \frac{dU}{dx} = \\ = \frac{\tau_s}{\rho U^2} - \frac{\tau_w}{\rho U^2} \end{aligned} \quad (139)$$

where

$$u_s = u \text{ at } y = s$$

$$\tau_s = \tau \text{ at } y = s.$$

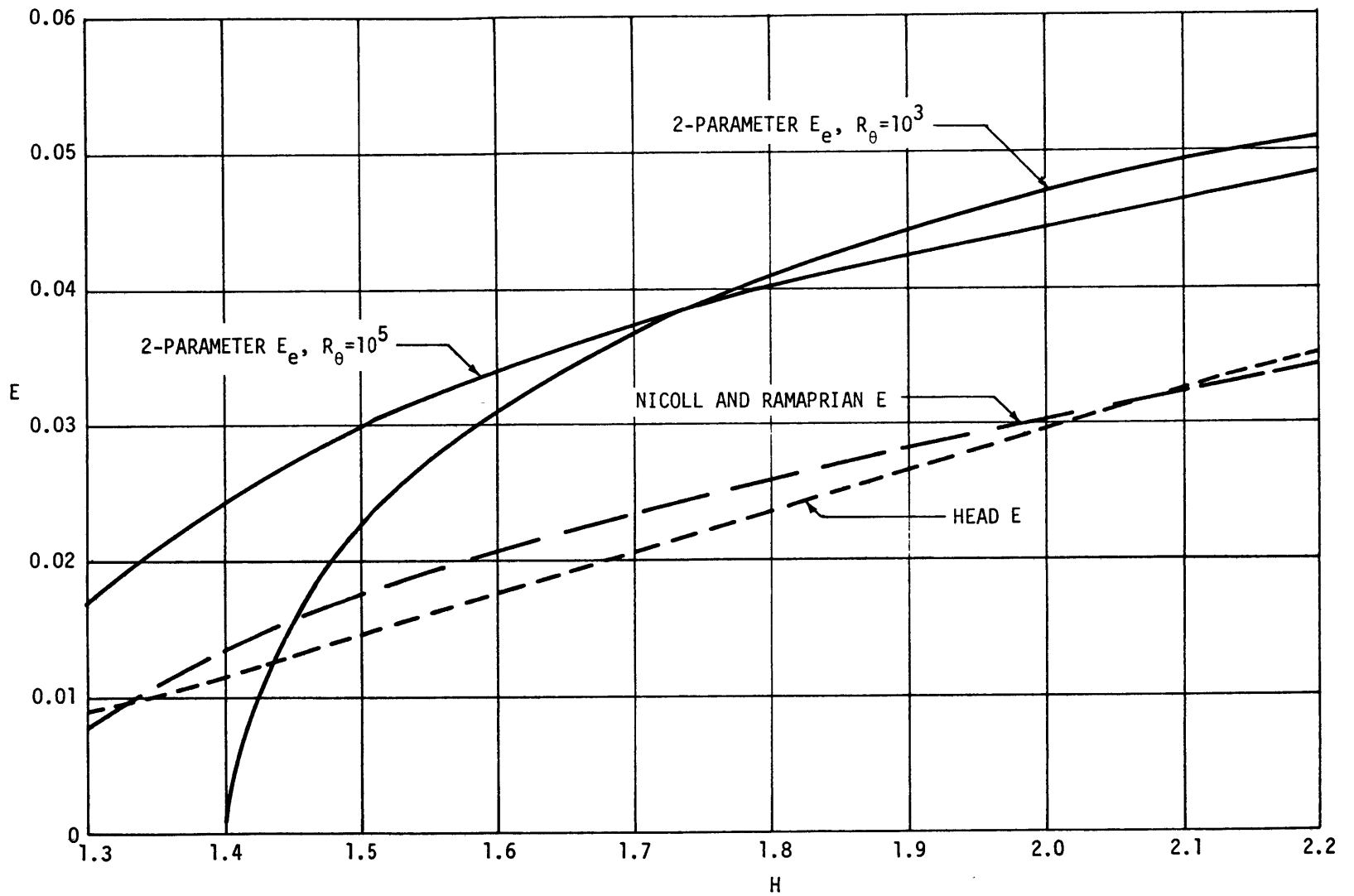


Figure 6 - Entrainment Method,  $E = f[H, R_\theta]$ ; Comparison of Various Procedures

Two new shape parameters are introduced:

$$\hat{H} \equiv \frac{\int_0^s \frac{u}{U} dy}{\theta} \quad (140)$$

and

$$\bar{H} \equiv \frac{\int_0^s \left(\frac{u}{U}\right)^2 dy}{\theta} \quad (141)$$

Then Equation (139) becomes

$$\begin{aligned} \frac{d}{dx} (\bar{H}\theta) - \frac{u_s}{U} \frac{d}{dx} (\hat{H}\theta) + \left(2\bar{H} - \frac{u_s}{U} \hat{H} - \frac{s}{\theta}\right) \frac{\theta}{U} \frac{dU}{dx} \\ = \frac{\tau_s}{\rho U^2} - \frac{\tau_w}{\rho U^2} \end{aligned} \quad (142)$$

With

$$\hat{H} = f[H, \ln R_\theta] \quad (143)$$

and

$$\bar{H} = f[H, \ln R_\theta] \quad (144)$$

$$\theta \frac{dH}{dx} = \left\{ \frac{H\bar{H} + (H+1) \left[ \frac{\partial \bar{H}}{\partial \ln R_\theta} - \frac{u_s}{U} \left( \hat{H} + \frac{\partial \hat{H}}{\partial \ln R_\theta} \right) \right] + \frac{s}{\theta}}{\frac{\partial \bar{H}}{\partial H} - \frac{u_s}{U} \frac{\partial \hat{H}}{\partial H}} \right\} \frac{\theta}{U} \frac{dU}{dx} \quad (145)$$

$$- \left[ \frac{\bar{H} + \frac{\partial \bar{H}}{\partial \ln R_\theta} - \frac{u_s}{U} \left( \hat{H} + \frac{\partial \hat{H}}{\partial \ln R_\theta} \right) + 1 - \frac{\tau_s}{\tau_w}}{\frac{\partial \bar{H}}{\partial H} - \frac{u_s}{U} \frac{\partial \hat{H}}{\partial H}} \right] \frac{\tau_w}{\rho U^2}$$

Then



$$M = - \frac{H\bar{H} + (H+1) \left[ \frac{\partial \bar{H}}{\partial \ln R_\theta} - \frac{u_s}{U} \left( \hat{H} + \frac{\partial \hat{H}}{\partial \ln R_\theta} \right) \right] + \frac{s}{\theta}}{\frac{\partial \bar{H}}{\partial H} - \frac{u_s}{U} \frac{\partial \hat{H}}{\partial H}} \quad (146)$$

$$N = - \frac{\bar{H} + \frac{\partial \bar{H}}{\partial \ln R_\theta} - \frac{u_s}{U} \left( \hat{H} + \frac{\partial \hat{H}}{\partial \ln R_\theta} \right) + 1}{\frac{\partial \bar{H}}{\partial H} - \frac{u_s}{U} \frac{\partial \hat{H}}{\partial H}} \quad (147)$$

$$P = - \left( \frac{\partial \bar{H}}{\partial H} - \frac{u_s}{U} \frac{\partial \hat{H}}{\partial H} \right)^{-1} \quad (148)$$

$$\hat{C}_S = \frac{\tau_s}{\tau_w} \quad (149)$$

POWER-LAW VELOCITY PROFILE ( $s = m\delta$ )

For power-law velocity profiles Equation (16)

$$\hat{H} = \frac{2H}{H-1} \left( \frac{s}{\delta} \right)^{\frac{H+1}{2}} \quad (150)$$

and for  $s = m\delta$

$$\hat{H} = \frac{2H}{H-1} m^{\frac{H+1}{2}} \quad (151)$$

and

$$\frac{d\hat{H}}{dH} = - \frac{m^{\frac{H+1}{2}}}{(H-1)^2} \left[ 2 - H(H-1) \ln m \right] \quad (152)$$

Since

$$\frac{u_s}{U} = \left( \frac{s}{\delta} \right)^{\frac{H-1}{2}} \quad (153)$$

$$\frac{u_s}{U} = m^{\frac{H-1}{2}} \quad (154)$$

For power-law velocity profiles

$$\bar{H} = \left( \frac{H+1}{H-1} \right) \left( \frac{s}{\delta} \right)^H \quad (155)$$

and for  $s = m\delta$

$$\bar{H} = \left( \frac{H+1}{H-1} \right) m^H \quad (156)$$

$$\frac{d\bar{H}}{dH} = - \frac{m^H}{(H-1)^2} \left[ 2 - (H^2-1) \ln m \right] \quad (157)$$

Therefore from Equations (146), (147), and (148)

$$M = \frac{H(H+1)}{\ln m} \left( 1 - m^{1-H} \right) \quad (158)$$

$$N = \frac{(H-1)}{\ln m} \left( 1 + m^{-H} \right) \quad (159)$$

$$P = - \frac{(H-1)}{m^H \ln m} \quad (160)$$

POWER-LAW VELOCITY PROFILE ( $s = \theta$ )

From Equations (150) and (19)

$$\hat{H} = \frac{2H}{(H-1)} \left[ \frac{H-1}{H(H+1)} \right]^{\frac{H+1}{2}} \quad (161)$$

and

$$\frac{d\hat{H}}{dH} = \left[ \frac{H-1}{H(H+1)} \right]^{\frac{H+1}{2}} \left\{ \frac{H}{H-1} \ln \left[ \frac{H-1}{H(H+1)} \right] - 1 \right\} \quad (162)$$

Also

$$\frac{u_s}{U} = \left[ \frac{H-1}{H(H+1)} \right]^{\frac{H-1}{2}} \quad (163)$$

$$\bar{H} = \frac{1}{H} \left[ \frac{H-1}{H(H+1)} \right]^{H-1} \quad (164)$$

and

$$\frac{d\bar{H}}{dH} = -\frac{1}{H} \left[ \frac{H-1}{H(H+1)} \right]^{H-1} \left\{ \frac{H-1}{H+1} - \ln \left[ \frac{H-1}{H(H+1)} \right] \right\} \quad (165)$$

Then

$$M = \frac{H(H+1) - (H-1) \left[ \frac{H-1}{H(H+1)} \right]^{-H}}{\ln \left[ \frac{H-1}{H(H+1)} \right]} \quad (166)$$

$$N = \frac{(H-1) \left\{ 1 - \left[ \frac{H-1}{H(H+1)} \right]^{-H} \right\}}{\ln \left[ \frac{H-1}{H(H+1)} \right]} \quad (167)$$

$$P = -\frac{H-1}{\left[ \frac{H-1}{H(H+1)} \right]^H \ln \left[ \frac{H-1}{H(H+1)} \right]} \quad (168)$$

TWO-PARAMETER VELOCITY PROFILE ( $s = m\delta$ )

The objective is to obtain  $\tilde{H}[H, R_\theta]$  and  $\bar{H}[H, R_\theta]$ . Let us introduce shape parameter  $\hat{G}$  defined as

$$\hat{G} \equiv \frac{\int_0^{s/\delta} \left( \frac{U-u}{u_\tau} \right) d\left(\frac{y}{\delta}\right)}{\int_0^1 \left( \frac{U-u}{u_\tau} \right) d\left(\frac{y}{\delta}\right)} \quad (169)$$

From appropriate definitions

$$\hat{H} = \left( \frac{\delta}{\theta} \right) \left( \frac{s}{\delta} \right) - H\hat{G} = \left( \tilde{H}+H \right) \left( \frac{s}{\delta} \right) - H\hat{G} \quad (170)$$

Then for  $s = m\delta$

$$\hat{H} = (\tilde{H}+H)m - H\hat{G} \quad (171)$$

Likewise let us introduce shape parameter  $\bar{G}$  defined as

$$\bar{G} \equiv \frac{\int_0^{s/\delta} \left( \frac{U-u}{u_\tau} \right)^2 d\left(\frac{y}{\delta}\right)}{\int_0^1 \left( \frac{U-u}{u_\tau} \right) d\left(\frac{y}{\delta}\right)} \quad (172)$$

From appropriate definitions

$$\bar{H} = (H-1) \frac{\bar{G}}{G} + \left(\frac{\delta}{\theta}\right) \left(\frac{s}{\delta}\right) - 2 \hat{H}G = (H-1) \frac{\bar{G}}{G} + (\tilde{H}+H) \frac{s}{\delta} - 2 \hat{H}G \quad (173)$$

For  $s = m\delta$

$$\bar{H} = (H-1) \frac{\bar{G}}{G} + (\tilde{H}+H)m - 2 \hat{H}G \quad (174)$$

From the law of the wake, Equation (12)

$$\hat{G} = \frac{A m(1-\ln m) + B_2 m \left[ 1 - m^2 \left( 1 - \frac{m}{2} \right) \right]}{A + \frac{1}{2} B_2} \quad (175)$$

$\hat{G}$  is related to  $G$  through  $B_2$  in Equation (23).

An empirical fit for  $m = 0.3$  ( $m = 0.3$  was used by Moses) gives

$$\hat{G}_{0.3} = 0.5541 + \frac{0.410}{G} \quad (176)$$

or from Equation (22)

$$\hat{G}_{0.3} = 0.5541 + 0.410 \frac{H}{(H-1)} \left( \frac{\tau_w}{\rho U^2} \right)^{\frac{1}{2}} \quad (177)$$

Unfortunately the condition  $s = \theta$  does not lend itself to this type of analysis.

Then

$$\hat{H}_{0.3} = 0.3(\tilde{H}+H) - H \left[ 0.5541 + \frac{0.410H}{H-1} \sqrt{\frac{\tau_w}{\rho U^2}} \right] \quad (178)$$

Since

$$I_1 \bar{G} = \int_0^m \left( \frac{U-u}{u_\tau} \right)^2 d\left(\frac{y}{\delta}\right)$$

consequently

$$\begin{aligned} I_1 \bar{G} &= A^2 m (\ln^2 m - 2 \ln m + 2) - 2 A B_2 m (\ln m - 1) \\ &+ A B_2 m^3 \left[ (2-m) \ln m + \frac{1}{4} m - \frac{2}{3} \right] \\ &+ B_2^2 m \left( 1 - 2m^2 + m^3 + \frac{9}{5} m^4 - 2 m^5 + \frac{4}{7} m^6 \right) \end{aligned} \quad (179)$$

For  $m = 0.3$

$$I_1 \bar{G}_{0.3} = 1.7573 A^2 + 1.2512 A B_2 + 0.2571 B_2^2 \quad (180)$$

$\bar{G}$  is related to  $G$  through  $B_2$  by Equation (23). An empirical fit gives for  $m = 0.3$

$$\frac{\bar{G}_{0.3}}{G} = 0.686 + \frac{2.04}{G^2} \quad (181)$$

or

$$\frac{\bar{G}_{0.3}}{G} = 0.686 + \frac{2.04 H^2}{(H-1)^2} \frac{\tau_w}{\rho U^2} \quad (182)$$

Then from Equation (174)

$$\bar{H}_{0.3} = 0.3 \bar{H} - 0.122 H - 0.686 - \frac{0.820 H^2}{H-1} \sqrt{\frac{\tau_w}{\rho U^2}} \left( 1 - 2.488 \sqrt{\frac{\tau_w}{\rho U^2}} \right) \quad (183)$$

From Equation (10) for  $m = s/\delta$

$$\frac{u_s}{U} = 1 - \sqrt{\frac{\tau_w}{\rho U^2}} \left[ -A \ln m + B_2 \left( 1 - \frac{1}{2} w[m] \right) \right] \quad (184)$$

A linearized fit of Equation (23) yields

$$B_2 = 1.364 (G - 4.78) \quad (185)$$

Then for  $m = 0.3$

$$\left(\frac{u}{U}\right)_{0.3} = 1 - 1.069 \frac{(H-1)}{H} + 2.232 \sqrt{\frac{\tau_w}{\rho U^2}} \quad (186)$$

Now from Equation (183)

$$\begin{aligned} \frac{\partial \bar{H}}{\partial H}_{0.3} = 0.3 \frac{\partial \tilde{H}}{\partial H} - 0.122 - \frac{0.820H}{(H-1)} \sqrt{\frac{\tau_w}{\rho U^2}} \left[ \frac{(H-2)}{(H-1)} \left( 1 - 2.488 \sqrt{\frac{\tau_w}{\rho U^2}} \right) \right. \\ \left. - \frac{2 c_2}{2.3(H_0+1)^\gamma} \left( 1 - 4.976 \sqrt{\frac{\tau_w}{\rho U^2}} \right) \right] \end{aligned} \quad (187)$$

and from Equation (178)

$$\frac{\partial \hat{H}}{\partial H}_{0.3} = 0.3 \frac{\partial \tilde{H}}{\partial H} - 0.2541 - \frac{0.410H}{H-1} \sqrt{\frac{\tau_w}{\rho U^2}} \left[ \frac{H-2}{H-1} - \frac{2 c_2}{2.3(H_0+1)^\gamma} \right] \quad (188)$$

Also

$$\begin{aligned} \frac{\partial \bar{H}}{\partial \ln R_\theta}_{0.3} = \frac{0.820H^2}{(H-1)} \sqrt{\frac{\tau_w}{\rho U^2}} \left[ 1 - 6.07 \sqrt{\frac{\tau_w}{\rho U^2}} - 2.08 \left(\frac{H}{H-1}\right)^{3/2} \left(\frac{\tau_w}{\rho U^2}\right)^{1/4} \right]_x \\ \times \left[ \frac{\alpha_1}{2.3} \left(\frac{\tau_w}{\rho U^2}\right)_0^{1/2} + \frac{2\alpha(H_0-1)^2}{2.3(H_0+1)} \left( \frac{c_2}{2.3H_0\gamma_0} - \frac{\ln \frac{\gamma}{\gamma_0}}{H_0+1} \right) \right] \end{aligned} \quad (189)$$

and

$$\begin{aligned} \frac{\partial \hat{H}}{\partial \ln R_\theta}_{0.3} = \frac{0.410H^2}{(H-1)} \sqrt{\frac{\tau_w}{\rho U^2}} \left[ 1 - 4.17 \left(\frac{H}{H-1}\right)^{3/2} \left(\frac{\tau_w}{\rho U^2}\right)^{1/4} \right]_x \\ \times \left[ \frac{\alpha_1}{2.3} \left(\frac{\tau_w}{\rho U^2}\right)_0^{1/2} + \frac{2\alpha(H_0-1)^2}{2.3(H_0+1)} \left( \frac{c_2}{2.3H_0\gamma_0} - \frac{\ln \frac{\gamma}{\gamma_0}}{H_0+1} \right) \right] \end{aligned} \quad (190)$$

MOMENT OF MOMENTUM METHOD

If the equation of motion, Equation (78), is multiplied by  $y$  and then integrated from  $y = 0$  to  $y = \delta$ , a moment of momentum equation is formed. Unfortunately the resulting equation is awkward to deal with on the basis of the two-parameter velocity profile in order to obtain a shape parameter equation. However a convenient form results from a power-law velocity profile which Tetervin and Lin<sup>31</sup> originally obtained

$$\theta \frac{dH}{dx} = - \frac{H(H+1)(H^2-1)}{2} \frac{\theta}{U} \frac{dU}{dx} + H (H^2-1) \frac{\tau_w}{\rho U^2} - \frac{(H+1)(H^2-1)}{2} C_\tau \quad (191)$$

where

$$C_\tau \equiv 2 \int_0^1 \frac{\tau}{\rho U^2} d\left(\frac{y}{\delta}\right) \quad (192)$$

is the shear-stress integral.

From Equation (83)

$$M = \frac{H(H+1)(H^2-1)}{2} \quad (193)$$

$$N = H H^2 - 1 \quad (194)$$

$$P = \frac{(H+1)(H^2-1)}{2} \quad (195)$$

Then from Equation (86)

$$C_{\tau,e} = \left\{ \beta + \frac{2H}{H+1} + \frac{2A[H+(H+1)\beta](H-1)}{(H+1)^2 [GH+A H-1]^2} \right\} \frac{\tau_w}{\rho U^2} \quad (196)$$

Nash and Hicks<sup>4</sup> use

$$C_{\tau,e} = 0.025 \left(1 - \frac{1}{H}\right)^2 \quad (197)$$

while Nash and Macdonald<sup>32</sup> suggest

$$C_{\tau,e} = (\beta + 1.16) \frac{\tau_w}{\rho U^2} \quad (198)$$

Also McDonald<sup>33</sup> proposes

$$C_{\tau,e} = (0.93 \beta + 1.2) \frac{\tau_w}{\rho U^2} \quad (199)$$

Values of  $C_{\tau}$  for strong adverse pressure gradients are given in Reference 12 as

$$C_{\tau} = \left( \frac{H-1}{H+1} \right)^2 \psi_o \quad (200)$$

where

$$\psi_o = \left( \frac{H_o}{H_o - 1} \right) \left[ 1 + \frac{0.0378 \sqrt{52.9 \log H_o - 4.18}}{H_o^2 - 1} \right] \left( \frac{\tau_w}{\rho U^2} \right)_o \quad (201)$$

Figure 7 compares the shear stress integrals of some of the various formulations.

#### NONEQUILIBRIUM PRESSURE GRADIENTS

The use of equilibrium stress factors not only ensures agreement for equilibrium pressure gradients but also for quasi-equilibrium conditions where the  $G$  values do not remain constant but vary in accordance with the equilibrium  $G$ - $\beta$  relation. Relating equilibrium stress factors to  $H$  and  $R_o$  provides a built-in lag which is characteristic of the response of the shear-stress distribution to sudden changes in pressure gradients. Lag-type equations have been proposed by Goldberg<sup>21</sup> and Nash and Hicks<sup>4</sup> of type

$$\delta \frac{d C_S}{dx} = \lambda (C_{S,e} - C_S) \quad (202)$$

where  $\lambda$  is a constant adjusted to suit the experimental data.



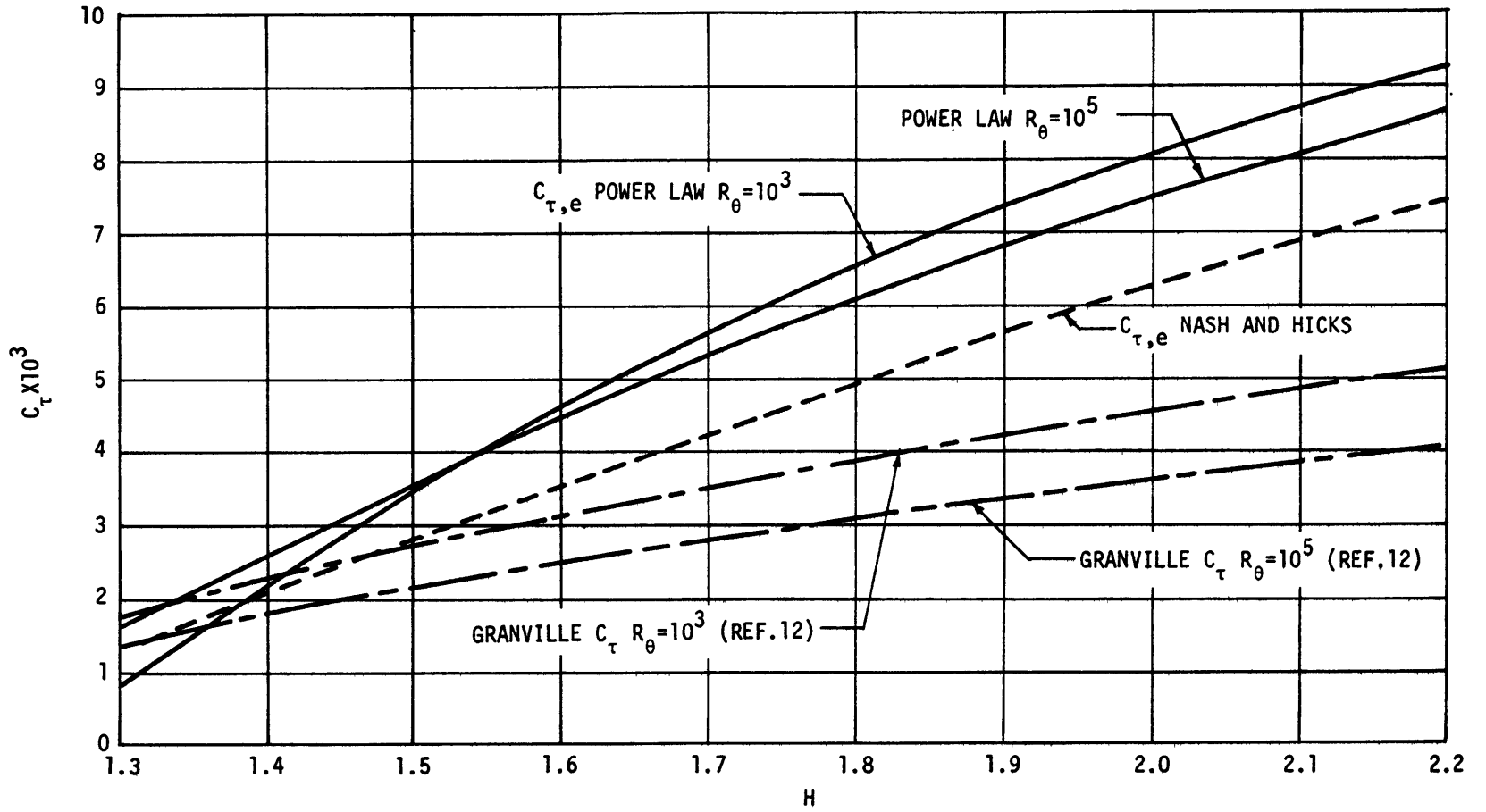


Figure 7 - Moment of Momentum Method,  $C_\tau = f[H, R_\theta]$ ; Comparison of Various Procedures

Close to separation ordinary boundary layer conditions seem to fail. There are three-dimensional cross flows, normal-stress effects and normal pressure variations. This region merits a study of its own.

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