

REPORT NO. 1541

A NUMERICAL SOLUTION FOR FLOW BETWEEN ROTATING AND STATIONARY FINITE DISKS

by

Donald H. McCoy

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June 1971

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BALLISTIC RESEARCH LABORATORIES

REPORT NO. 1541

JUNE 1971

A NUMERICAL SOLUTION FOR FLOW BETWEEN ROTATING AND STATIONARY FINITE DISKS

Donald H. McCoy

Exterior Ballistics Laboratory

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ABERDEEN PROVING GROUND, MARYLAND

BALLISTIC RESEARCH LABORATORIES

REPORT NO. 1541

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A NUMERICAL SOLUTION FOR FLOW BETWEEN ROTATING AND STATIONARY FINITE DISKS

ABSTRACT

The velocity field and pressure distribution between stationary and rotating finite disks are obtained by a finite difference solution of the Navier-Stokes equations and the continuity equation using an explicit scheme developed by Chorin [1]. These numerical results are compared with the results of "infinite disk" solutions. In the range of interest for viscometer applications, the velocity and pressure are adequately represented by the infinite disk solutions.

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TABLE OF SYMBOLS

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Symbol	Definition	
D	Artificial density	
N	Number of iterations	
P	Pressure	
P	Dimensionless pressure, D/8	
(r, 0, z)	Cylindrical coordinates	
(R, 8, Z)	Dimensionless cylindrical coordinates	
R	Maximum radial distance	
Re	Reynolds number	
t	Time	
T	Dimensionless time	
u	Radial velocity	
U	Dimensionless radial velocity	
U _{co}	Dimensionless radial velocity from	
	the infinite disk solution	

7

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Tangential velocity

Dimensionless tangential velocity

Axial velocity

v

V

W

W

Z

ð

λ

μ

V

P

Q

Dimensionless axial velocity

Maximum axial distance

Artificial compressibility factor

Ratio, Z/R SIDELDIS

Viscosity using extensionsmid

Kinematic viscosity, " µ/porily"

(R, B, C) Dimensionless cylinan **Density**

Angular velocity ibsr menaixeM

Reynolds number

Time

Symbol

D

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U

TY

Dimensionless time

Radial velocity

itmensionless radial velocity

Dimensionless radial vefacity from the infinite disk solution

I. INTRODUCTION

The problem of practical interest to be solved can be stated as follows: An incompressible Newtonian liquid is contained between an upper, stationary disk and a lower, rotating disk. Determine the velocity field and pressure distribution of the fluid in motion.



By presenting accurate numerical solutions to the Navier-Stokes equations, this paper makes possible an effective determination of the nature of inertial secondary motions in a rotating disk viscometer having disks of finite radii. Previous considerations of the problem have assumed that the disks are of infinite radii, resulting in solutions involving numerical and analytical approximations [2, 3]. A comparison will be made of the velocity distributions obtained with disks of finite radii and those having radii assumed to be infinite.

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II. THE EQUATIONS OF MOTION

The equations governing flow consist of those of Navier-Stokes that the particle of effective terms is moldom off together with the equation for continuity. These, as used in this paper, are for a cylindrical coordinate system (r, θ , z) with rotabe thread to be a term of the state state (received a term be defined as be defined as and (received a term tional symmetry and incompressibility assumed. The equations are defined and a term of the state being trace we written as follows [4]:

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial r} - \frac{v^2}{r} + w \frac{\partial u}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial r} + v \left[\frac{\partial^2 u}{\partial r^2} + \frac{1}{r} \frac{\partial u}{\partial r} - \frac{u}{r^2} + \frac{\partial^2 u}{\partial z^2} \right]$$
(1)

$$\frac{\partial \mathbf{v}}{\partial t} + \mathbf{u} \frac{\partial \mathbf{v}}{\partial \mathbf{r}} + \frac{\mathbf{u}\mathbf{v}}{\mathbf{r}} + \mathbf{w} \frac{\partial \mathbf{v}}{\partial \mathbf{z}} = \mathbf{v} \left[\frac{\partial^2 \mathbf{v}}{\partial \mathbf{r}^2} + \frac{1}{\mathbf{r}} \frac{\partial \mathbf{v}}{\partial \mathbf{r}} - \frac{\mathbf{v}}{\mathbf{r}^2} + \frac{\partial^2 \mathbf{v}}{\partial \mathbf{z}^2} \right]$$
(2)

$$\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial r} + w \frac{\partial w}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \left[\frac{\partial^2 w}{\partial r^2} + \frac{1}{r} \frac{\partial w}{\partial r} + \frac{\partial^2 w}{\partial z^2} \right]$$
(3)

cutations, this paper makes possible an offertive determination of the

(4) nature of idential secondary motions in a relating $d = \frac{w \delta}{z \delta} + \frac{u \delta}{r \delta}$

Here, u, v, and w represent, respectively, velocities in the Here, u, v, and w represent, respectively, velocities in the base assumed that the disks are of the end to be realify in solution radial, tangential, and axial directions. ρ is density; p is pressure; radial, tangential, and axial directions. ρ is density; p is pressure; and ρ is kinematic viscosity. The radius of each disk is R; the to be and a solution of the velocity direction of the viscos of the solution of the viscosity of the rotating spacing between them is \overline{Z} ; and the angular velocity of the rotating the viscosity of the rotating of the velocity of the rotating the viscosity of the rotating of the velocity of the rotating the viscosity of the rotating viscosity of the rotating the viscosity of the rotating viscosity of the rotating viscosity of the rotating viscosity of viscosity viscosity of viscosity viscos

disk is Q.

The above equations are normalized according to the following procedure:

Primary Variable	Normalizing Quantity	Normalized Variable
u, v, w	RΩ	U, V, W
r	R	R
z .	Z	Z
· • •	Z/RO	Т
. Р	ρR ² Ω ²	P
The normalize	d equations are	Ť
$\frac{\partial \mathbf{U}}{\partial \mathbf{T}} + \lambda \left(\mathbf{U} \ \frac{\partial \mathbf{U}}{\partial \mathbf{R}} - \frac{\mathbf{V}^2}{\mathbf{R}} \right)$	$) + W \frac{\partial U}{\partial Z} = -\lambda \frac{\partial P}{\partial R}$	
Estimaty one at $Z = 1$. $\frac{1}{R}$	$- \left[\lambda^{2} \left(\frac{\partial^{2} U}{\partial R^{2}} + \frac{1}{R} \frac{\partial U}{\partial R} - \frac{U}{R^{2}} \right]$	$\left[\frac{\partial^2 U}{\partial Z^2}\right]$ (5)
$\frac{\partial \mathbf{V}}{\partial \mathbf{T}} + \lambda \left(\mathbf{U} \frac{\partial \mathbf{V}}{\partial \mathbf{R}} + \frac{\mathbf{U}\mathbf{V}}{\mathbf{R}} \right)$	+ W $\frac{\partial V}{\partial Z} = \frac{1}{R_e} \left[\lambda^2 \left(\frac{\partial^2 V}{\partial R^2} + \frac{1}{R} \right) \right]$	$\frac{\partial \mathbf{V}}{\partial \mathbf{R}} - \frac{\mathbf{V}}{\mathbf{R}^2} + \frac{\partial^2 \mathbf{V}}{\partial \mathbf{Z}^2} \right] (6)$
$\frac{\partial W}{\partial T} + \lambda U \frac{\partial W}{\partial R} + W \frac{\partial W}{\partial Z}$	$\frac{V}{d} = -\frac{\partial P}{\partial Z} + \frac{1}{R_e} \left[\lambda^2 \left(\frac{\partial^2 W}{\partial R^2} + \frac{1}{2} \right) \right]$	$\frac{1}{R}\frac{\partial W}{\partial R} + \frac{\partial^2 W}{\partial Z^2} \right] $ (7)
$\lambda\left(\frac{U}{R}+\frac{\partial U}{\partial R}\right)+\frac{\partial W}{\partial Z}$,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,	n mer sogeralismer (8)
where $R_e = \frac{\overline{ZR\Omega}}{\nu}$	and $\lambda = \overline{Z/R}$.	instant official as

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painello The boundary conditions used are shown in Figure 2.

procedure:

U = V = W = 0 U = V = W = 0 U = 0 U = 0 U = 0 U = 0 U = 0 U = 0 U = 0 U = 0 U = 0 U = 0 U = 0 U = 0

Figure 2. Boundary Conditions

 $\frac{40}{38} \lambda = \frac{10}{56} \frac{1}{10} + \frac{10}{10} \frac{1}{10} - \frac{10}{10} \frac{1}{10} \frac{1}{10} \frac{1}{10} + \frac{10}{10} \frac{1}{10} \frac{1}$

Lyinary Varmul Normal sine C.

The rotating plate is at Z = 0 and the stationary one at Z = 1. R = 0 is the axis of rotation and R = 1 is the liquid-air interface which

is assumed to be planar.

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(3)

(1)

 (Θ)

ALLE LAY LA THE TAK

The Reynolds Number, R_e , represents the measure of inertial stresses relative to viscous stresses. Viscometers are usually analyzed by considering the "inertialess" limit of $R_e = 0$, in which case the flow consists of circular streamlines. Viscometers generally operate with a gap-to-radius ratio, λ , of less than 0.05 and with the product λR_e less than unity.

III. NUMERICAL PROCEDURE

 $U(T^*) = (-1) U(T^*) = U(R^*) - U(R^*)$

To solve the Navier-Stokes equations, Chorin's explicit technique is used. An artificial compressibility factor, δ , is introduced into the equations of motion. Hence the continuity equation is replaced by

$$\frac{\partial D}{\partial T} = -\lambda \left(\frac{U}{R} + \frac{\partial U}{\partial R} \right) - \frac{\partial W}{\partial Z}$$
(9)

where $P = D/\delta$ is an artificial equation of state.

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Equations (5), (6), (7), and (9) are put into finite difference form by means of a Dufort-Frankel scheme, (details are obtained from Chorin [1]), and the resultant equations appearing below are then utilized in the computing procedure:

13

$U(T^{+}) = \left\{ -\frac{\lambda \Delta T}{\Delta R} U [U(R^{+}) - U(R^{-})] \right\}$

TIL NUMEROLALITEOREMUNT

- for the large structure + 2 $\lambda \Delta T$, $\frac{V^2}{R}$ - solution to the large of

beoutorin at a construction free produced in the abeer af orpin

$-\frac{\Delta T}{\Delta Z} = \frac{T \Delta T}{U (z^+) - U (z^-)}$

placed by

$\frac{\lambda \Delta T}{\delta \Delta R} \left[D(R^{+}) - D(R^{-}) \right]$

where $P = D/\delta$ is an interval equation of state.

$+ \frac{2\lambda^{3}\Delta T}{R\Delta R^{3}} [U(R^{+}) + U(R^{-}) - U(T^{-})]$

by means of a Putert-Freetel's heme, (details are obtained from Chorte [1]). $\frac{1}{2} \left(\frac{-R}{R} - \frac{1}{2} \frac{1}{2} \frac{U_0}{R} \frac{1}{R} \frac{1}{2} \frac{S_0}{R} \frac{1}{R} \frac{1}{2} \frac{S_0}{R} \frac{1}{R} \frac{1}{2} \frac{1$

11.7.

velized in the computing procedure:

$\frac{2\lambda^{2}\Delta T}{R} = \frac{U}{R^{2}}$

14

+ $\frac{2\Delta T}{R_{e}\Delta Z^{2}}$ [U(Z⁺) + U(Z⁻) - U(T⁻)]

+ U (T⁻)
$$\Big/ \Big\{ 1 + \frac{2\lambda^2 \Delta T}{R \Delta R^2} + \frac{2\Delta T}{R \Delta Z^2} \Big\}$$
 (10)

$$\mathbf{V}(\mathbf{T^+}) = \left\{ -\frac{\lambda \Delta \mathbf{T}}{\Delta \mathbf{R}} \mathbf{U} \left[\mathbf{V} \left(\mathbf{R^+} \right) - \mathbf{V} \left(\mathbf{R^-} \right) \right] \right\}$$

$$-2\lambda\Delta T \frac{UV}{R}$$

$$-\frac{\Delta T}{\Delta Z} W [V(Z^+) - V(Z^-)]$$

+
$$\frac{2\lambda^{3}\Delta T}{R_{e}\Delta R^{2}}$$
 [V(R⁺) + V(R⁻) - V(T⁻)]

$$+\frac{\lambda^{2} \Delta T}{R \Delta R} \begin{bmatrix} V(R^{+}) - V(R^{-}) \\ R \end{bmatrix}$$

$$-\frac{2\lambda^2 \Delta T}{R_e} \frac{V}{R^2}$$

$$+\frac{2\Delta T}{R_{e}\Delta Z^{2}} [V(Z^{+}) + V(Z^{-}) - V(T^{-})]$$

+ V (T⁻)
$$\Big\} / \Big\{ 1 + \frac{2\lambda^2 \Delta T}{R_e \Delta R^2} + \frac{2\Delta T}{R_e \Delta Z^2} \Big\}$$
 (11)

W (T⁺) =
$$\left\{ -\frac{\lambda \Delta T}{\Delta R} \right\}$$
 U [W (R⁺) - W (R⁻)] + T)

$$-\frac{\Delta T}{\Delta Z} W [W(Z^+) - W(Z^-)]$$

$$-\frac{\Delta T}{\delta \Delta Z} \left[D(Z^+) - D(Z^-) \right]$$

+
$$\frac{2\lambda^{3} \Delta T}{R_{e} \Delta R^{3}} [W(R^{+}) + W(R^{-}) - W(T^{-})]$$

$$+ \frac{\lambda^2 \Delta T}{R_e \Delta R} \begin{bmatrix} W (R^+) - W (R^-) \end{bmatrix}$$

+
$$\frac{2\Delta T}{R\Delta Z^2}$$
 [w(z⁺) + w(z⁻) - w(T⁻)]

$$+ W (T^{-}) \Big\} / \Big\{ 1 + \frac{2\lambda^{3} \Delta T}{R_{e} \Delta R^{3}} + \frac{2\Delta T}{R_{e} \Delta Z^{2}} \Big\}$$
(12)

16

 $+ v(\tau^{-})] \land \left\{1 + \frac{2\lambda^2 \Delta T}{R_2 \Delta R} + \frac{2\Delta T}{R_2 \Delta R} + \frac{2\Delta T}{R_2 \Delta R} \right\}$

(1.1)

$$D(T^+) = -2\lambda \Delta T \frac{U}{R}$$

$$-\frac{\lambda \Delta T}{\Delta R} \left[U \left(R^+ \right) - U \left(R^- \right) \right]$$

$$-\frac{\Delta T}{\Delta Z} [W(Z^+) - W(Z^-)] + D(T^-)$$
(13)

where $U(R^+) = U(R + \Delta R, Z, T);$ $U(R^-) = U(R - \Delta R, Z, T);$ etc.

Equations (10) through (13) are applicable only at interior points. For the various boundaries, the following equations were developed using for the purpose one-sided differences.

At the centerline, (R = 0), U = 0 (14) V = 0 (15) W (T⁺) = $\left\{ -\frac{\Delta T}{\Delta Z} W \left[W (Z^+) - W (Z^-) \right] - \frac{\Delta T}{\delta \Delta Z} \left[D (Z^+) - D (Z^-) \right] + \frac{2\lambda^2 \Delta T}{R_e \Delta R^2} \left[2 W (R^+) - W (T^-) \right] \right\}$

+ $\frac{2\Delta T}{R_{\Delta}Z^{2}}$ [W (Z⁺) + W (Z⁻) - W (T⁻)]

+ W (T⁻) } / { 1 + $\frac{2\lambda^2 \Delta T}{R_e \Delta R^2}$ + $\frac{2\Delta T}{R_e \Delta Z^2}$ } (16)

 $D(T^{+}) = -\frac{2\lambda\Delta T}{\Delta R} U(R^{+})$

(11)

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The state of the s

 $- \frac{\Delta T}{\Delta Z} [W (Z^+) - W (Z^-)] + D (T^-).$ (17)

Equations (10) through (13) are applie able only at interior

At the liquid-air interface, (R = 1),

(18)developed using for the purpose one-eth-d differen0 = U

 $\mathbf{V}(\mathbf{T}^+) = \left\{ -\frac{\Delta \mathbf{T}}{\Delta \mathbf{Z}} \quad \mathbf{W} \left[\mathbf{V} \left(\mathbf{Z}^+ \right) - \mathbf{V} \left(\mathbf{Z}^- \right) \right] \right\} \text{ solution}$

+ $\frac{\lambda^{3} \Delta T}{R \Delta R^{3}}$ [2 V (R⁻⁻) - 4 (R⁻) + V (T⁻)]

 $W(T^{+}) = \{-\frac{\Delta T}{\Delta Z}, W(W(S^{+}) - W(Z^{-}))\}$

+ $\frac{2\Delta T}{R_{\Delta}Z^{2}}$ [V(Z⁺) + V(Z⁻) - V(T⁻)]

 $-\frac{\Delta T}{\delta \Delta Z}$ (D(2+) - D(2-))

 $+ V (T^{-}) \bigg\} / \bigg\{ 1 - \frac{\lambda^{2} \Delta T}{R \Delta R^{2}} + \frac{2 \Delta T}{R \Delta Z^{2}} \bigg\}$ (19)

$$W(T^+) = \left\{ -\frac{\Delta T}{\Delta Z} W \left[W(Z^+) + W(Z^-) \right] \right\}$$

•

•

•

•

$$-\frac{\Delta T}{\delta \Delta Z} \left[D(Z^+) - D(Z^-) \right]$$

+
$$\frac{\lambda^{2} \Delta T}{R_{e} \Delta R^{2}}$$
 [2 W (R⁻⁻) - 4 W (R⁻) + W (T⁻)]

+
$$\frac{2 \Delta T}{R_e \Delta Z^2}$$
 [W (Z⁺) + W (Z⁻) - W (T⁻)]

+ W (T⁻)
$$\left\{ 1 - \frac{\lambda^2 \Delta T}{R_e \Delta R^2} + \frac{2 \Delta T}{R_e \Delta Z^2} \right\}$$
 (20)

$$D(T^+) = \frac{2\lambda\Delta T}{\Delta R} U(R^-)$$

$$-\frac{\Delta T}{\Delta Z} [W(Z^{+}) - W(Z^{-})] + D(T^{-}). \qquad (21)$$

Along the rotating disk, (Z = 0),

- U = 0 . (22)
- $\mathbf{V} = \mathbf{R} \tag{23}$
- $W = 0 \tag{24}$

$$D(T^{+}) = (-32 \frac{\Delta T}{\Delta Z} [W(Z^{+}) W(Z)] + D(T^{-})_{W}$$
(25)

Along the stationary disk (Z = 1),

U =

(-T)

1051

(15)

(SS)

(25)

(15)

$$\frac{1}{6} \left[\frac{1}{2} \left(\frac{1}{2} \right) - \frac{1}{2} \left(\frac{1}{2$$

$$D(T^{+}) = -2 \frac{\Delta T}{\Delta Z} [W(Z) - W(Z^{-})] + D(T^{-}).$$
(29)
(-T) W - (-Z) W + (+Z) W] $\frac{T\Delta S}{S_{2}} +$

The above equations (to obtain steady state solutions) were programmed for a digital computer. Such solutions are considered to have been achieved when $D(T^+)$ and $D(T^-)$ are approximately equal. A copy of the computer program, coded in Fortran IV, is contained in Appendix A.

$$-\frac{\Delta T}{2\pi} (W(Z^{+}) - W(Z^{-})] + D(T^{-}).$$

Along the rotating disk, (Z = 0),

V. FRICAL RESELLS AND DISCUSSION

IV. STABILITY

the second by flow found in the K-Z plane are shown for

If the flow under consideration is to remain incompressible

and therefore stable, Chorin asserts that δ must be chosen so that

$\int_{\frac{1}{2}}^{\frac{1}{2}} R_{e}^{2} Max \left(U^{2} + V^{3} + W^{2} \right)^{\frac{1}{2}} < 1.$

traveldes it empresses, Zerodo era Thees from much

This is equivalent to keeping the artificial Mach Number less than dominant of contact 1 A base dominant of unity.

To insure that motion can be adequately described in the confines of a given grid scheme ΔT must be chosen so that

> $\Delta T \leq .437 \text{ Min } (\Delta R \text{ or } \Delta Z) \delta^{\frac{1}{2}}.$ R.Z¹⁰⁰

> > e 00

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Numerical results show that the above conditions are not sufficient for stability in some cases and too restrictive in others.

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V. NUMERICAL RESULTS AND DISCUSSION

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The secondary flows found in the R-Z plane are shown for $\lambda = 1.0, .50, .20$ and .10 in Figures 3 through 6. ($\lambda R_e = 1.0$ in each case.) Computer program outputs for velocities and pressures corresponding to these cases are given in Appendix B.

Once an adequate δ and ΔT are chosen, convergence is achieved in 2000 to 4000 iterations. This requires 20 to 40 minutes of comand additional dock that intra and gargest of molecuppe at end T puter time. The following table contains δ and ΔT values for which .vtime divergence occurs:

- (<u>3</u> 13-1)	di . hoo	tropolo vi	atsipabs :	na aso natiom	tedt stuart Chor	in's Conditions
	λ γ	n us seec R e	a dirial.	<u>values</u> mailos <u>A</u> T	ô	<u>AT</u>
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Figure 3. Secondary Velocity Field

bias with the left and $(\lambda = 1.0, R_e = 1.0)$

12.5 .

11 16

Scale: .25" = .00458



Z = 0.0

Figure 3. Secondary Velocity Field

(Figure 4. Secondary Velocity Field

 $(\lambda = .5, R_e = 2.0)$ Scale: .25" = .00636



Figure 5. Secondary Velocity Field ($\lambda = .2, R_e = 5.0$)

Scale: . 25" = . 00677



Secondary Velocity Field

Scale: . 25" = . 00740

Scale: . 25" - . 00677

Listed below are δ and ΔT values for which there are convergent

solutions:

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				Chorin	s
		Trial	Values	Necessary C	onditions
ð	Re	5	ΔΤ	δ	ΔT
1.0	1.0	. 04	.00025	1.0	. 022
. 75	1.0	. 04	.00075	1.0	. 022
. 75	1.0	. 04	.00075	1.0	. 022
. 75	1.0	. 09	. 00075	1.0	. 022
. 75	1.0	. 16	.0005	1.0	. 022
. 75	1.0	. 16	. 00075	1.0	. 022
. 75	10.0	.04	.0005	. 01	. 002 *
. 50	1.0	. 09	.001	1.0	. 022
. 50	2.0	.04	.001	. 25	.011
. 20	1.0	. 09	. 001	1.0	. 022
. 20	5.0	. 04	.0005	. 04	.004
. 20	5.0	. 04	.001	. 04	.004
. 10	1.0	. 04	.001	1.0	. 022
. 10	10.0	. 04	.001	· . 01	. 002 *
. 10	5. 0	. 04	.001	. 04	. 004

* The necessary conditions are violated, yet a convergent solution is obtained.

At least 20 grid spaces are necessary in R and Z for reasonable results. Numerical oscillations are produced in the Z direction when only ten grid spaces are used.

The share

For the small secondary flows, true convergence is very difficult to discern with this numerical technique. As an example, for $\lambda = .1$ and $R_e = 5.0$ convergence seems to have been achieved in 2000 iterations. But when the results are plotted, numerical oscillations in U are evident. However, after 4000 iterations these oscillations have vanished and the results appear to be accurate when compared with the infinite disk solution.

Rate of convergence is independent of starting values unless accurate values at R = 1.0 for W are introduced. In proof of this, a case was run which took as an initial state the infinite disk solution. Convergence still required 2000 iterations.

In order to test the stability of the numerical solution, two special computer runs were made. In the first, a convergent solution was perturbed in W. This perturbation was of the order Max |W| and R,Z was introduced between R = 0.5 and 0.6. The solution again converged, suggesting stability for finite perturbations. For the second run, initial values were assumed from the infinite disk solution and pressure was increased by an order of magnitude. This solution achieved an oscillatory behavior then diverged completely.

VI. CONCLUSIONS

A large scale parametric examination of δ , ΔT , ΔR and ΔZ is required to best determine the numerical stability, the rate of convergence, and the accuracy of each problem solved using Chorin's technique. To show that the results obtained herein are indeed reasonable a numerical solution is compared with a series solution for infinite disks. The series solutions are written to second order in λR as follows [2, 3]:

$$\dot{U} = -R \lambda R_{e} [4 (1-Z) - 9 (1-Z)^{2} + 5 (1-Z)^{4}]$$

 $V = R (1-Z) - \frac{R\lambda^2 R^2}{e} [-8 (1-Z) - 35 (1-Z)^4 + 63 (1-Z)^6] - 20 (1-Z)^7]$

$$W = \frac{\lambda R}{60} \left[-4 (1-Z)^{2} + 6 (1-Z)^{3} - 2 (1-Z)^{5} \right]$$

 $P = .15 R^{2}$.

These comparisons are made for $\lambda = .1$ and R = 5.0. Figure 7 shows the secondary velocity field for the numerical solution. (A computer output for this solution is contained in Appendix B.)

L = 1.0
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technique. To show that the resolto obtained introis are indo d
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$V = R (1-Z) - R \lambda^{2} R \frac{2}{6300} \left[-8 (1-Z) - 35 (1-Z)^{2} + 63 (1-Z)^{2} \right] - 60$
$\begin{bmatrix} a(Q-1) & S & - & Q = 0 & 0 \\ (Q-1) & S & - & (Q-1) & 0 + & M & (Q-1) & 0 \end{bmatrix} = \begin{bmatrix} A & K & = & W \\ A & M & M & M \end{bmatrix}$

Figure 7. Secondary Velocity Field

 $(\lambda = .1, R_e = 5.)$

Scale: .25" = .00369

These comparisons are made for $\lambda = .1$ and R = 5.0. Figure 1 shows the secondary velocity field for the numerical solution. (A computer output for this solution is contained in Appendix B.)

Figure 8 shows a graphical representation of a comparison between the numerical solution for finite disks and the series solution for infinite disks.



Considering the closeness of the results of the above comparison it can definitely be stated that the velocity and pressure for rotating diskviscometer applications are adequately represented by the series solution of the infinite disk problem. Figure 8 shows a graphical representation is conjunate when the numerical solution for figure doke and the certe volution for the infinite disks and the certe volution for the infinite disks.

ACKNOWLEDGEMENTS

Grateful acknowledgement is made to Professor Morton M. Denn for his assistance and encouragement, to Monte Coleman for assistance with CALCOMP Plotter programming, to Muriel Ewing for helpful suggestions in preparing the manuscript, and to Alcise Beatty for the excellent typing.

Figure 8. . . 8 arogiT

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Considering the closeness of the results of the above comparisen it can definitely be stated that the valority and promove to rotating disk viscometer applications are adequately represented by the series solution of the infinite disk problem.

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APPENDIX A

- Charm, Alexan**MARDORP RETURNOD** Encompressible Viscour Else Problem Computational Physics Vol. 2, No. 1, Aug. 1995, and 17-20.
 - 2. Mellor, G. L.; Chapple, P. J.; and tokes, V. K. "On Play Between a Rotating and a "tational Biese, "Journal of Flaid Mechanics, Vol. 31, Part 1, 1968, pp. 83-112

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1. Factor all constants and remove their computation from the innermost integration loop. This results in a 30%

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2. Simplify the subscript arithmetic as much as possible.
(See statement number 5.) This results in a 20% reduction of computer time.

DIMENSION U(51,21,3),V(51,21,3),W(51,21,3),D(51,21,3),R(51),Z(21) 57 おうかくして、まやバネスラジーへのをまとりとまるのうみ DIV=1.0+2.0+DT+(LAM2/DR2+1.0/D22)/RE 01V1=1.0+DT#(-LAM2/DR2+2.0/DZ2)/RE READ(5,51)LAM.RE.DEL.DT.DR.DZ.L.M 中国の日本日本日本 「「「「」」 R([+])=R(])+DR Z0+(I)Z=(I+I)Z 00 20 I=1, MM1 H([,J,K)=0.0 VI0/0-1=VI0X U(I,J,K)=0.0 V(I,J,K)=0.0 V(I,1,K)=R[I] 3. 1-1,L M D(I, J, K)=0.0 HRITE(6,52) LAN2=LAM++2 K=1,3 - H-1-0 D0 3 K=1,3 J. 1=1. DR2=DR*+2 C2=02++2 R(1)=0. -0=1 2 REAL LAM I-H-IHH ZCL 1 IIZ 20 00 00 20 20

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U([JJI:3]=(-U(IJ.1,2)+(U(IJ+1,1,2)-U(IJ-1,1,2))+C1 \$+{U(13+1,1,2)+U(13-1,1,2)-U(13,1,1))+C5 5+{U(13P+1,2)+U(13N,1,2)-U(13,1,1))+C8 -W([J,1,2)+(U([JP,1,2)-U([JN,1,2))+C3 FVASSOUS+ FOLDISIVE C4=CAM+07A(DR+DEL) \0x5+5*0\0551\x6 1+C6+(U([J+1,1,2)-U([J-1,1,2))/R([]) -(0f1J+1,1,2)-0(1J-1,1,2))+C4 C7+0113+1+2)/R110+2, *DK *D5 C5=2.0+LAN2+DT/(RE+DR2) \$+62+V([J,1,2)++2/R([) C10=LAM2+DT/(RE+DR2) C6-LAM2+DT/(RE+DR) C8-2.0+01/(RE+022) C7=2.0+LAM2+DT/RE C13=2.0+LAN+DT/DR C11=01/(DEL+02) C9+DT/(DEL+DR) RDIV1-1-0/DIV1 C12=2.0+DT/DZ TC+(I) J+CNPCI C2=2.0+LAN+DT C3=07/D2 5-0+8 SC+1 I+NEN=N C1=LAM+DT/DR TNN+2=6 9 00 MA I+d[N=d[] 15+(2-7)-NON DO 6 [=Z+LM] NJ=(1-1)+51 NUP=Jest

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V(L,J,3)=((2.0*V(L-2,J,2)-4.0*V(L-1,J,2)+V(L,J,1))*C10 V([J,1,3)=(-U[],1,2)+(V([J+1,1,2)-V([J-1,1,2))+C] w([],],],3)=(-U([],1,2)*(W([J+1,1,2)-W([J-1,1,2))*C] W(1,J,3)=(-W(1,J,2)*(W(1,J+1,2)-W(1,J-1,2))*C3 \$+{V[1]+1,1,2)+V[1]-1,1,2)-V([],1,1))#C5 6+(M(IJ+1,1,2)+W(IJ-1,1,2)-W(IJ,1,1))+C5 5-W([J,1,2)*(V([JP,1,2)-V([JN,1,2))*C3 \$+(V(IJP,1,2)+V(IJN,1,2)-V(IJ,1,1))*C8 i-W(IJ.1,2)+(W(IJP,1,2)-W(IJN,1,2))+C3 5+ (W(IJP,1,2)+W(IJN,1,2)-W(IJ,1,1))*C8 i-(M(IJP,1,2)-W(IJN,1,2))*C3+D(IJ,1,1) b-W(L, J, 2) * (V(L, J+1, 2)-V(L, J-1, 2)) *C3 i+(V(L, J+1, 2)+V(L, J-1, 2)-V(L, J, 1))*C8 6+C6*(W(IJ+1,1,2)-W(IJ-1,1,2))/R(1) \$+C6*(V([J+1,1,2)-V([J-1,1,2))/R([) i-(U([J+1,1,2)-U([J-1,1,2))*C1 6-C2*U(IJ,1,2)*V(IJ,1,2)/R(I) \$-{D(1,]+1,2)-D(1,]-1,2))*C11 D([],],])=-C2*U([],],2)/K([) i-(D(IJP,1,2)-D(IJN,1,2))+C9 \$+(2.0*H(2,J,2)-H(1.J,1))*C5 6-C7+V([J,1,2)/R([)++2 1101+1(1,1,1,1))+RDIV 5+U([J,1,1,])*RDIV 5+V(IJ.1.1))*RDIV +V(L,J,1))*RDIVI DO 7 J=2, MM1 CONTINUE

\$+(K(1,J+1,2)+K(1,J-1,2)-F(1,J,1))+C8

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WRITE(6,51)LAM,RE,DEL,DT,DR,DZ,L,M HRITE(6,50)[V([,J,3],[=],L,LMT] ₩RITE(6,50)(U(I,J,3),I=1,L,LMT) J=J-JMT WRITE(6,50)(W(1,J,3),I=1,L,LMT) CMAX=AMAX1 (DMAX, ABS (D(1, 3,3))) IF (MOD(N, 100) .NE.0)GOTO 5 IF(DMAX.GT.1.0)G0 T0 101 le. D(1,J,2)=D(1,J,3) D(I.J.1)=D(I.J.2) H[1.J.])=W[1.J.2] IF(J.6T.0)GOT0 12 IF(J.GT.0)GOT0 11 DO 100 J=1.M WRITE(6,53)N D0 100 I=1.L WRITE(6,54) WRITE(6,52) JHT=MM1/10 WRITE(6,55) LMT=LM1/10 WRITE(6.52) WRITE(6,56) CONTINUE DMAX=0.0 エピーフェフ 1+N=N ビニフ 100 101 10 12 13

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J=J-JMT IF(J.GT-0) MRITE(6,52 MRITE(6,52 J=M WRITE(6,50	IFUJ-67-0) IFUN-LT-40 60T0 1 STOP FORMATCLE	FORMAT(// FORMAT(2H) FORMAT(2H) FORMAT(2H)	FORMAT CHART	Correction (SILLING SILLING SILLING
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SAMPLE COMPUTER OUTPUTS

APPENDIX B

Guide to the computer output:

- 1. The first line of print is the number of iterations.
- 2. The second line of print is λ , R_e, δ , ΔT , ΔR , ΔZ and a pair of indices.
- 3. The velocity fields (U, V, W) and the density $(D = \delta P)$ are printed on the following lines. The numbers are oriented

so as to correspond to the graphs previously shown.

N= 4000

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